

(19)



Europäisches Patentamt
European Patent Office
Office européen des brevets



(11) Publication number:

0 280 127 B1

(12)

EUROPEAN PATENT SPECIFICATION

(45) Date of publication of patent specification: **31.07.91** (51) Int. Cl.⁵: **B03B 4/00**

(21) Application number: **88102053.1**

(22) Date of filing: **12.02.88**

(54) **Pneumodensitometric vibrating separator for sorting of heterogeneous bulk products.**

(30) Priority: **27.02.87 IT 1242887**

(43) Date of publication of application:
31.08.88 Bulletin 88/35

(45) Publication of the grant of the patent:
31.07.91 Bulletin 91/31

(84) Designated Contracting States:
AT BE CH DE ES FR GB GR LI LU NL SE

(56) References cited:
EP-A- 0 081 072
EP-A- 0 159 050
EP-A- 0 162 014
EP-A- 0 247 423
DE-A- 3 606 238

(73) Proprietor: **OFFICINE MECCANICHE FERRERO S.p.A.**

Via Privata Trento n. 4
I-17047 Vado Ligure (Sv)(IT)

(72) Inventor: **Ferrero, Francesco**
Via Privata Trento n. 4
I-17047 Vado Ligure(IT)

(74) Representative: **Ferrarotti, Giovanni**
Studio Consulenza Tecnica Dr. Ing. Giovanni
Ferrarotti Via Alla Porta Degli Archi 1/7
I-16121 Genova(IT)

EP 0 280 127 B1

Note: Within nine months from the publication of the mention of the grant of the European patent, any person may give notice to the European Patent Office of opposition to the European patent granted. Notice of opposition shall be filed in a written reasoned statement. It shall not be deemed to have been filed until the opposition fee has been paid (Art. 99(1) European patent convention).

Description

This invention covers a pneumodensitometric vibrating separator table for separation of heterogeneous bulk products, in particular compost, i.e. to separate fertilizers obtained through aerobic biotransformation of putrescible organic solid domestic wastefractions, from other foreign matter not yet kept back during previous sorting.

Pneumodensitometric vibrating separators are known in which the vibrating or oscillating table, provided with a permeable bottom or bed through which air flows so as to form an air stream fluidizing the heterogenous matter to be sorted out which is located above this air stream. Usually, this vibrating or oscillating motion is unidirectional according to pulses having a 20° - 30° inclination with respect to the longitudinal sorting plane, this inclination being based upon the material to be treated; furthermore, the inclination angle of the separator table has the same direction as the vibration or oscillation pulse angle.

The heterogeneous material on the table is thus separated by gravity due to the upwards penetrating air stream; the particulate matter having a higher specific gravity will tend to move downwards in touch with the table, while those with a lower specific gravity will tend to float.

This means that the heavier components in touch with the table are exposed to the direct vibration or oscillation of the table and will move towards the top of the table where they are discharged, whereas lighter items will move downwards to be discharged there.

A vibrating separator as described is known by EP-A-0081072 in which it is substantially foreseen an inclined, oscillating and permeable table, on which the heterogeneous material is separated by gravity and by air stream fluidizing.

This document aims to better the separation adopting on the table transversal plates, useful to detain the heterogeneous material over the heavier components.

Besides there are known as technological background EP-A-0247423, DE-A-3606238, EP-A-0162014 and EP-A-0159050.

These known pneumodensitometric tables for separation of heterogeneous bulk products, operating according to the above mentioned principle, have however various drawbacks which make them little selective and inefficient.

A first drawback is caused by the fact that existing pneumodensitometric tables have a rather uneven vibrating or oscillating motion because of their poorly secured, scantily stretched, flat perforated bottom. As a result, the perforated plate will be subject to deformation, particularly during use, losing its stiffness and uniform movement, nega-

tively affecting the sorting operation.

A second drawback is due to the vibration and oscillation characteristics as currently used, which do not allow for adequate separation of very dense, fine particle layers such as for instance glass splinters, sand, porcelain or earthenware chips, small pieces of rigid plastic material etc.

Another disadvantage lies in the fact that it is impossible with existing pneumodensitometric separators to change the direction of vibration or oscillation so that the operating conditions cannot be adjusted to the nature of the material to be treated.

The non uniform fluidizing action of the air stream passing through the perforated bed is another drawback. Existing tables are in porous material (felt, unwoven fabric, perforated plate, wire cloth etc.) all having the aim to minimize air flow resistance for obvious energy saving reasons. At the same time, the pressure has to be reduced also because of the above mentioned mechanical fastening defects of the table. The rate of discharge will therefore vary according to the density of the material, since the air tends to penetrate through the thinner part of the layer and this may even cause the material upwards to be hurled upwards while resulting of course in an irregular and uneven fluidizing process, which jeopardizes the efficiency of the equipment.

This patent has the aim to eliminate all above listed drawbacks of the known pneumodensitometric separators and to optimize sorting of heterogeneous bulk material. According to this Patent, the fluidizing bed is centered and is resting on intermediate supports.

It is kept stretched by appropriate devices located on the side frames. In this way, the table is well secured and the vibratory motion over its whole surface is perfectly uniform. According to the patent, the vibratory or oscillating motion is directly and evenly transmitted to the sorting plane by means of stiffening side frames.

According to the Patent, a rather high vibration frequency with low amplitude has been chosen so as to boost separation of even small sized high density matter. The unidirectional vibratory motion is indeed involving 1000 to 3000 vibrations per minute with a 0.5-4 mm amplitude.

Furthermore, according to the Patent, unidirectional vibration is achieved by two vibrators or rotary propelling devices having controlled eccentric masses generating centrifugal forces by their identical and coplanar opposed rotation. The propelling assembly is rigidly connected to the side frames by means of sturdy links so that the two equal and opposed centrifugal forces, when combined, will provide the unidirectional reciprocating force necessary to impress the required motion. The system connecting the propelling devices to

the side frames permits to vary the angle α resulting from the direction of the vibration force and the longitudinal plane of the separator and this system is also equipped with longitudinal conveyance facilities, so that the unidirectional vibratory forces are always passing through the center of gravity of the machine. It follows that the movement of the pneumodensitometric table can be adjusted based upon the quality of the material to be classified, in order to ensure its operation in optimum conditions.

According to the Patent, the drawback regarding the non uniform fluidizing action of the air stream is obviated by using a special sandwich panel as a permeable bed, the top of which is consisting of a closely perforated metal sheet (with closely-spaced holes having a diameter ranging between 0.5 and 0.6 mm), while the bottom is consisting of a high strength but low permeability filter fabric (nylon, polyester or polyamide, etc.). This sandwich panel will require a high air generator pressure (about 450-550 mm w.c.) to obtain an adequate air flow for fluidizing of the material, while the down flow will be virtually independent from the height of the top layer of the material.

Furthermore, the Patent also eliminates the trouble caused by the fact that dust raised during fluidizing of the material and now collected by dust collectors has to be conveyed to cyclones and separate dust catching filters. Indeed, in this Patent, the dust collecting system is an integrating part of the separator and is consisting of a filter mounted on the pneumodensitometric table so as to create a plenum chamber from which coarser dust particles will fall back onto the table, whereas the finer particles are held back by the filter, there to felt up or agglomerate, until they are periodically returned to the fluidizing bed. This greatly simplifies dust catching equipment since a separate delivery system with all related handling problems is no longer necessary.

According to the Patent, the system also includes facilities for processing of special materials containing shreds of plastics or other fibrous fabrics which tend to flutter during fluidizing.

As already explained before, this problem is particularly felt when separating compost. To eliminate this trouble, a closed loop nylon or wire gauze belt or strip is mounted on two rolls or drums, between the table and the dust catcher. This rotating belt will catch these fluttering particles adhering to its lower leg, discharging them at one end by gravity and with the aid of special devices.

In subordinate, the pneumodynamometric Table may perform an oscillating movement, i.e. a low frequency reciprocating movement (150-450 cycles/minute and high amplitude ranging between 10 and 40 mm).

The invention in question is illustrated in its

practical and exemplifying implementation in the enclosed drawings, in which:

Fig. 1 shows a cross section of the pneumodensitometric table;

Fig. 2 shows a side view of the pneumodensitometric table illustrated in Fig. 1;

Fig. 3 shows in detail how the porous panel is connected to one side frame;

Fig. 4 shows a side view of a separator, complete with dust catcher;

Fig. 5 shows a side view of the separator illustrated in Fig. 4, fitted with a dust catcher for air entrained particles;

Fig. 6 shows a side view of the discharging end of the dust collecting belt.

With reference to these Figures: the porous sandwich panel 1 is consisting of a closely perforated metal top sheet 2 and in a bottom filtering cloth 3 in moisture resisting synthetic fabric.

This porous panel 1 is placed transversally, in a centered position, properly shimmed 4, resting on the frame of the separator so as to ensure a stable configuration. The flanged panel edges are braced by stirrups 5 and turnbuckles 6 mounted on the side frames 7. This ensures a stable position of the perforated panel and its uniform vibratory motion. Furthermore, this sandwich panel has high strength and low permeability with respect to the fluidizing air flow.

A closed chamber 8, is communicating at its lower end by means of an elastic coupling 9, with a high pressure fan 10 pressing the air through the porous panel.

The unidirectional vibratory motion applied to the pneumodensitometric table by two electric rotary vibrators 11 having eccentric masses, mounted at the opposite ends of a tubular cross member 12 of great thickness, so that the two axes of rotation are parallel and coplanar while the two equal and combined counter-rotating centrifugal forces are resulting in an unidirectional reciprocating force necessary for the vibratory motion of the separator.

The tubular cross member 12, bearing the vibrators 11, is supported by U bolts 13 which can be moved along the sections 14, rigidly connected to the side frames 7.

The vibrators 11 can rotate on the axis of the cross member 12 so that it will be possible to adjust the inclination angle α of the vibratory directions with respect to the longitudinal plane of the table 1 and the vibrators 11 can also be moved lengthwise forwards and backwards on the sections 14, so that the unidirectional vibrating force will pass through the center of gravity G of the vibrating part of the machine. This longitudinal traverse of the cross member 12 can be achieved, for example, by providing the section 14 with numerous holes in which the threaded ends of the U

bolts 13 can be fitted.

The pneumodensitometric table is supported by an appropriate mounting or frame 15 by means of flexible devices 16 such as cables, ropes, chains etc. secured to hooks 17 on the side frames 7.

These flexible supporting means 16 are adjustable to ensure uniform distribution of the weight and position control of the table at its optimum longitudinal inclination angle α' with respect to the horizontal plane.

This angle α' has the same direction as the angle α of the vibration forces with respect to the table plane. Hence, the unidirectional vibratory force has an $\alpha + \alpha'$ inclination with respect to the horizontal plane.

An outlet 18 for the heavier particles B is provided in the upper section of the table while another outlet 19 in the lower table section discharges the finer and lighter particles C. The outlet 19 is fitted with an adjustable diaphragm 19' determining the layer thickness of the material on the table. A feeder trough 10 delivers the product A on the pneumodensitometric table 1 for selection.

The above explanations clearly illustrate the operation cycle. The product A to be sorted out is fed on the pneumodensitometric table 1 where it is fluidized by the air passing through the perforated plate. This will cause the broad or flat shaped light weight material having a low specific gravity, to be air entrained, without being effected by the vibratory impulses exerted on the table, while material having a high specific gravity will tend to settle on the vibrating surface and is thus moved forward under these pulse conditions.

Therefore, the lighter particles will flow downward (outlet 19) whereas the heavier products will move upwards towards the sorting table (outlet 18).

According to this invention a dust filter chamber 21 is located above the mounting 15 and is secured to the side frames of the pneumodensitometric table by means of a bellows 22. This filter chamber 21 is housing cloth tube filters 21'. Dust is sucked through these cloth tubes by means of a suction fan 23.

This configuration permits to combine the vibrating separator and dust filter in one single structure, thus offering additional functional advantages, since a spacious plenum chamber is available between the pneumodensitometric table and the dust filter, which is very useful for collection of coarse dust particles. Furthermore, any dust settling externally on the cloth tube filter 21' will tend to agglomerate or felt and can be periodically discharged once more on the vibrating table. This will make a separate dust selection unnecessary, since the dust is mixed with the lighter products C.

If the material to be sorted is not only containing hard and dense items (glass, stones, metals,

pottery fragments etc.) but also shreds of air entrained plastic material, lint and the like, these would hinder efficient sorting and would rapidly clog the dust filters without adequate recovery. To obviate these drawbacks, this patent presents a handy facility for separation and recovery of such air entrained waste.

This equipment is illustrated in Fig. n° 5 and n° 6 and is consisting of a loop shaped wire gauze or nylon belt 24 moving on two rolls or drums 25, 26, one of which 26 is motor-driven, acting as a tension roller and located at the back of the machine. This horizontal belt is intersecting the plenum chamber above the pneumodensitometric table and its lower leg moves towards the external roll 26.

The shreds, entrained by the air flowing towards the dust filters, will settle on the lower leg of the belt 24 and are discharged outside the machine. Since the air is not flowing beyond the wall diaphragm 27, the discharged matter falls into a dust hopper D.

Falling is facilitated by the air diffuser 28 detaching the air entrained particles from the belts, while a rotary brush 29 will further clean the wire gauze 24.

The lower outlet threshold 30 can be moved backward so that the air will prevalingly flow in direction of the arrow x with respect to y, to prevent the dust suction fan from generating excessive indraught in the machine which might detach the air entrained particles from the belt or prevent them from being externally discharged.

By this solution, the agglomerated dust when falling from the filter 21, will initially drop onto the upper belt section 24, to be conveyed to the roll 25 and from there dropped onto the pneumodensitometric table 1, to be subsequently discharged together with the lighter particles.

Based upon this description, the separator has therefore excellent operating features for sorting and classifying of heterogeneous material of any nature whatsoever; it has a built-in dust filter without separate dust recovery and also permits separation of air entrained matter which would be difficult to recover with existing separators.

Claims

1. Vibrating pneumodensitometric separator for sorting and removal of heterogeneous bulk material, consisting of a vibrating or oscillating densitometric table equipped with a porous bed for air fluidizing of the product, the air being separated by a fan (10) while the table (1) is moved by an unidirectionally vibratory action at an angle α with respect to the table, the latter's longitudinal inclination angle α' hav-

ing the same direction as the angle α , , so that this unidirectional vibratory action has an inclination angle $\alpha + \alpha'$ with respect to the horizontal plane and including dust catching facilities to collect the dust entrained by the fluidizing air stream, **characterized** by the following combined features:

- a) the porous table (1) is sandwich shaped, the upper layer (2) being in closely perforated metal sheet whereas the lower layer (3) is in moisture resisting man-made fabric; the table is transversally centered and supported by proper shims (4); it is strongly and evenly braced along its side frames (7) by tensioning devices (5, 6) so as to ensure great stiffness and low fluidizing air permeability,
- b) the vibratory motion is generated by two rotary electrovibrators (11) having counter-rotating eccentric masses which are secured at the opposite ends of a sturdy tubular cross member (12) so that the two vibrators are coplanar and parallel and so that the two equal and opposed combined centrifugal actions will result in an unidirectional reciprocating force having an inclination angle (α) with respect to the longitudinal table plane,
- c) table (1) is mounted by means of deformable devices (16) on a frame (15), these deformable devices being lengthwise adjustable for a uniform positioning of the table and to permit variation of its inclination angle (α');
- d) the cross member (12) bearing at its ends the electrovibrators (11) is secured to the side frames (7) of the table (1) by means of suitable devices (13, 14) which permit adjustment of the vibration force angle (α) with respect to the longitudinal table plane as well as lengthwise shifting of this cross member (12) and its electrovibrators (12) so that the vibratory force will pass through the center of gravity (G) of the vibrating part of the machine;
- e) a dust filter (21) complete with suction fan (21) and filter hoses (21') is mounted on the table frame (15), the filter (21) being connected to the side frames (7) of the table by means of bellows (22) delimiting a plenum chamber in which the coarse dust particles will fall so as to allow agglomeration of the collected dust which, after felling, will drop onto the table to be discharged together with the light-weight product (C) at the lower table end, thus eliminating the need for separate dust recovery;
- f) a special device to collect air entrained

material such as plastic films, paper, sheets, lint etc. is lodged inside the plenum chamber between the table (1) and the filter (21); this device is consisting of a closed loop wire gauze belt (24) wound around two rolls (25, 26), the external roll (26) being motor-driven and having a mobile lower leg running towards the external roll (26) so that the air entrained matter will stick to the lower belt leg because of the lifting force of the air flowing from the table (1) to the filter (21) then to be conveyed outside the machine and discharged by means of an air jet (28) and a rotary cleaning brush (29).

2. Separator as described in claim 1, **characterized** by the fact that the tensioning devices of the porous table (1) are stirrups (5) and turnbuckles (6) fitted on the side frames (7) and hooked to the flanged edges of the table (1).
3. Separator as described in claim 1, **characterized** by the fact that the devices securing the tubular cross member (12) bearing the electrovibrators (11) to the side frames (7) of the table (1) are U bolts (13) lengthwise movable on section shapes (14) which, in turn, are secured to the side frames (7) so as to permit variation of the inclination angle (α) of the unidirectional vibratory force with respect to the longitudinal table plane, while this lengthwise movement will ensure that this unidirectional vibratory force is always passing through the center of gravity (G) of the vibrating machine section.
4. Separator as described in claim 1, **characterized** by the fact that the movement generated by the electrovibrators (11) having eccentric masses, is a high frequency unidirectional vibratory motion of small amplitude ranging between 0.4 and 0.5 mm.
5. Separator as described in claim 1, **characterized** by the fact that the electrofan (10) generating the fluidizing air has a very high head of about 450-500 mm water column.
6. Separator as described in claim 1, **characterized** by the fact that the lower outlet threshold (30) is positioned backward and can be adjusted so that the air flowing through the belt will prevail on the air stream following the belt, so as to prevent indraught air from hindering conveyance of the air entrained matter sticking to the lower belt leg (24).
7. Separator as described in claim 1, **character-**

ized by the fact that the separator can be driven by a reciprocating traverse oscillation at low frequency ranging between 150 and 450 cycles per minute and a 10-40 mm high amplitude range.

Revendications

1. Séparateur vibrant de tri par densité pour trier et éliminer des produits hétérogènes en vrac, comprenant une table de tri par densité vibrante ou oscillante, équipée d'un lit poreux pour la fluidification des produits par l'air, l'air étant appliqué par un ventilateur (10) tandis que la table (1) est déplacée par une action vibratoire monodirectionnelle orientée sous un angle α par rapport à la table, l'angle d'inclinaison longitudinale α' de cette dernière étant le même que l'angle α , de sorte que l'action vibratoire monodirectionnelle est inclinée d'un angle $\alpha + \alpha'$ par rapport au plan horizontal, et comprenant des dispositifs collecteurs de poussière qui recueillent la poussière entraînée par le courant d'air de fluidification, caractérisé par les dispositions associées suivantes:

- a) la table poreuse (1) est une structure en sandwich, dont la couche supérieure (2) est une tôle métallique à perforations rapprochées, tandis que la couche inférieure (3) est en tissu synthétique résistant à l'humidité; la table est centrée dans la direction transversale et supportée par des cales appropriées (4); elle est solidement et régulièrement maintenue le long de ses bâtis latéraux (7) par des dispositifs tendeurs (5, 6) qui garantissent une grande rigidité et une faible perméabilité à l'air de fluidification,
- b) le mouvement vibratoire est créé par deux vibreurs électriques tournants (11) ayant des masses excentriques tournant en sens inverse qui sont montés aux extrémités opposées d'une robuste traverse tubulaire (12) afin que les deux vibreurs soient coplanaires et parallèles et que les deux actions centrifuges égales et opposées aient pour résultante une force alternative monodirectionnelle faisant un angle d'inclinaison (α) par rapport au plan de la table longitudinale,
- c) la table (1) est montée au moyen de dispositifs déformables (16) sur un bâti (15), ces dispositifs déformables étant réglables dans la direction longitudinale pour obtenir un positionnement uniforme de la table et permettre de faire varier son angle d'inclinaison (α');
- d) la traverse (12) portant à ses extrémités les vibreurs électriques (11), est maintenue

aux bâtis latéraux (7) de la table (1) par des dispositifs appropriés (13,14) qui permettent de régler l'angle (α) de la force vibratoire par rapport au plan de la table longitudinale également de décaler dans la direction longitudinale cette traverse (12) et ses vibreurs électriques (12) afin que la force vibratoire passe par le centre de gravité (G) de la partie vibrante de la machine ;

e) un filtre à poussière (21) complet avec un ventilateur d'aspiration (21) et des gaines filtrantes (21'), est monté sur le bâti de la table (15), le filtre (21) étant relié aux bâtis latéraux (7) de la table au moyen d'un soufflet (22) délimitant une chambre de distribution dans laquelle tombent les grosses particules de poussière en permettant ainsi l'agglomération de la poussière recueillie, laquelle, après s'être amalgamée, va tomber sur la table pour être déchargée en même temps que les produits de faible densité (C) à l'extrémité inférieure de la table, en évitant ainsi la nécessité d'un récupérateur de poussière distinct ;

f) un dispositif spécial pour collecter les matières entraînées par l'air telles que des films de matière plastique, du papier, des feuilles, des peluches, et analogues, qui sont logés à l'intérieur de la chambre de distribution entre la table (1) et le filtre (21); ce dispositif comprenant une courroie (24) bouclée en toile métallique passant autour de deux rouleaux (25, 26), le rouleau extérieur (26) étant entraîné par un moteur et le brin mobile inférieur de la courroie se déplaçant en direction du rouleau extérieur (26) afin que les produits entraînés par l'air se collent au brin inférieur de la courroie sous l'effet de la force de soulèvement de l'air allant de la table (1) vers le filtre (21), et soient ensuite entraînés à l'extérieur de la machine et sortent sous l'effet d'un jet d'air (28) et d'une brosse tournante de nettoyage (29).

2. Séparateur selon la revendication 1, caractérisé par le fait que les dispositifs tendeurs de la table poreuse (1) sont des étriers (5) et des tendeurs (6) fixés aux bâtis latéraux (7) et accrochés au rebord de la table (1).

3. Séparateur selon la revendication 1, caractérisé par le fait que les dispositifs maintenant la traverse tubulaire (12) qui supporte les vibreurs électriques (11) sur les bâtis latéraux (7) de la table (1) sont des étriers (13) mobiles dans la direction longitudinale sur des parties profilées (14) qui, à leur tour, sont fixés sur les bâtis

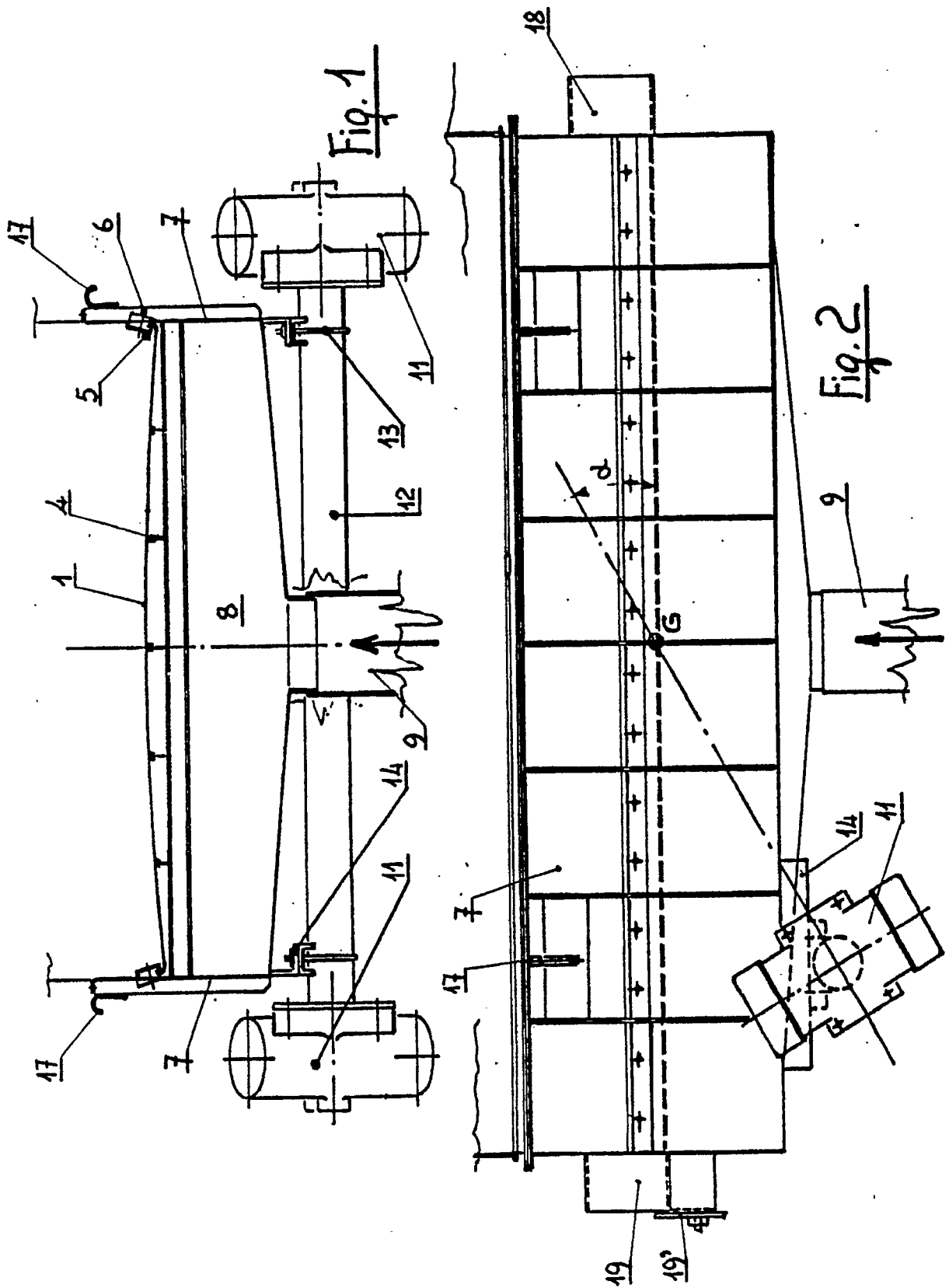
- latéraux (7) de façon à permettre de faire varier l'angle d'inclinaison (α) de la force vibratoire monodirectionnelle par rapport au plan de la table longitudinal, tandis que ce mouvement longitudinal garantit que la force vibratoire monodirectionnelle passe toujours par le centre de gravité (G) de la partie vibratoire de la machine. 5
4. Séparation selon la revendication 1, caractérisé par le fait que le mouvement créé par les vibreurs électriques (11) comportant des masses excentriques, est un mouvement vibratoire monodirectionnel à haute fréquence ayant une amplitude faible comprise entre 0,4 et 0,5 mm. 10 15
5. Séparateur selon la revendication 1, caractérisé par le fait que le ventilateur électrique (10) établissant le courant d'air de fluidification, exerce une pression très élevée d'environ 450 à 500 mm de colonne d'eau. 20
6. Séparateur selon la revendication 1, caractérisé par le fait que le seuil de sortie inférieure (30) est placé en arrière et peut être réglé afin que l'air traversant la courroie suive principalement la courroie, en évitant ainsi à l'air entraîné à l'intérieur de nuire à l'acheminement des matières portées par l'air et collant sur le bras inférieur de la courroie (24). 25 30
7. Séparateur selon la revendication 1, caractérisé par le fait que le séparateur peut être entraîné par une oscillation transversale alternative à une fréquence basse comprise entre 150 et 400 cycles par minute avec une amplitude élevée de 10 à 40 mm. 35

Patentansprüche

1. Vibrierender Windsichter zur dichteabhängigen Trennung und Entnahme von heterogenen Schüttgütern, bestehend aus einem vibrierenden oder hin- und hergehenden Tisch, der mit einem Bett zur Fluidifizierung des Gutes mittels Luft ausgestattet ist, wobei die Luft durch ein Gebläse (10) abgegeben wird, während der Tisch (1) durch eine einseitige Vibrationswirkung unter einem Winkel α in bezug auf den Tisch bewegt wird, und der Längsneigungswinkel α' die gleiche Richtung wie der Winkel α besitzt, so daß die einseitige Vibrationswirkung einen Neigungswinkel $\alpha + \alpha'$ in bezug auf die horizontale Ebene aufweist, wobei ferner Einrichtungen zum Sammeln des vom Luftstrom mitgenommenen Staubes vorgesehen sind, **gekennzeichnet** durch die folgenden Kombinationsmerkmale: 40 45 50 55

- a) Der poröse Tisch 1 ist aus Schichten aufgebaut, wobei die obere Schicht (2) aus einer eng perforierten Metallplatte und die untere Schicht (3) aus einem feuchtigkeitsbeständigen künstlichen Bespannstoff besteht; der Tisch ist durch geeignete Beilagen (4) quer zentriert und gelagert; er ist entlang seiner Seitengestelle (7) fest und gleichmäßig durch Spannvorrichtungen (5, 6) abgestützt, sodaß eine große Steifigkeit und eine geringe Durchlässigkeit für fluidifizierende Luft gesichert ist,
- b) die Vibrationsbewegung wird durch zwei, mit gegenläufig rotierenden exzentrischen Massen ausgestatteten drehbare Elektrovibratoren (11) erzeugt, die an den entgegengesetzten Enden eines kräftigen rohrförmigen Quergliedes (12) befestigt sind, sodaß die beiden Vibratoren in einer Ebene und parallel liegen und die beiden gleichen Zentrifugalwirkungen eine in einer Richtung wirkende hin- und hergehende Kraft hervorrufen, die in bezug auf die Tischlängsebene einen Neigungswinkel (α) besitzt,
- c) der Tisch (1) ist mittels verformbarer Mittel (16) an einem Rahmen (15) befestigt, wobei diese verformbaren Mittel der Länge nach einstellbar sind um eine gleichmäßige Positionierung des Tisches zu erreichen und eine Veränderung seines Neigungswinkels α' zu ermöglichen;
- d) Das an seinen Enden die Elektrovibratoren lagernde Querglied (12) ist an den Seitengestellen (7) des Tisches (1) mittels geeigneter Vorrichtungen (13,14) befestigt, die eine Veränderung des Winkels α der Vibrationskraft in bezug auf die Längsebene des Tisches und auch eine Längsverschiebung dieses Quergliedes (12) und seiner Elektrovibratoren (11) gestatten, sodaß die Vibrationskraft durch den Schwerpunkt (G) des vibrierenden Teiles der Maschine verläuft;
- e) Ein Staubfilter (21) mit einem Saugzuggebläse (23) und Filterschläuchen (21') ist am Tischrahmen (15) befestigt, wobei das Filter (21) mit den Seitengestellen (7) des Tisches mittels eines Balges (22) verbunden ist, wodurch eine Kammer begrenzt wird, in der die groben Staubpartikel herabfallen, um eine Zusammenballung des gesammelten Staubes zu ermöglichen, der auf den Tisch fällt, um zusammen mit dem leichtgewichtigen Gut (C) am unteren Tischende ausgetragen zu werden, sodaß die Notwendigkeit einer eigenen Staubentnahme entfällt;
- f) Eine besondere Vorrichtung zum Sammeln von von der Luft mitgenommenem Material,

- z.B. Kunststofffilme, Papier, Blätter, Zupfleinwand etc., ist innerhalb der Kammer zwischen dem Tisch (1) und dem Filter (21) angeordnet; diese Vorrichtung besteht aus einem geschlossenen Drahtgewebeband (24), das über zwei Rollen (25, 26) läuft, wobei die äußere Rolle (26) angetrieben ist und der untere Teil des Bandes sich gegen die äußere Rolle (26) bewegt, sodaß sich die von der Luft mitgenommenen Dinge zufolge der Hubkraft der von Tisch (1) zum Filter (21) strömenden Luft an das untere Band anlegen, um nach außerhalb der Maschine transportiert und mittels eines Luftstromes (28) und einer drehenden Reinigungsbürste (29) entladen zu werden.
2. Windsichter nach Anspruch 1, **dadurch gekennzeichnet**, daß die Spannvorrichtungen des porösen Tisches (1) aus Bügeln (5) und Spannschrauben (6) bestehen, die an den Seitengestellen (7) befestigt und in die mit Flanschen versehenen Kanten des Tisches (1) eingehakt sind.
3. Windsichter nach Anspruch 1, **dadurch gekennzeichnet**, daß die Vorrichtungen, die die Elektrovibratoren (11) lagernden rohrförmigen Querglieder (12) an den Seitengestellen (7) des Tisches (1) befestigen aus U-Bolzen (13) bestehen, die der Länge nach an Profilen (14) beweglich sind, die ihrerseits an den Seitengestellen (7) befestigt sind, sodaß eine Veränderung des Neigungswinkels α der in einer Richtung wirkenden Vibrationskraft in bezug auf die Längsebene des Tisches möglich ist, wobei diese Längsbewegung sichert, daß die Vibrationskraft immer durch den Schwerpunkt (G) des vibrierenden Teiles geht.
4. Windsichter nach Anspruch 1, **dadurch gekennzeichnet**, daß die, durch die mit exzentrischen Massen versehenen Elektrovibratoren (11) erzeugte Bewegung eine in einer Richtung wirkende Vibrationsbewegung hoher Frequenz und einer kleinen Amplitude im Bereich zwischen 0,4 und 0,5 mm ist.
5. windsichter nach Anspruch 1, **dadurch gekennzeichnet**, daß das Elektrogebläse 10, das die fluidifizierende Luft erzeugt, eine sehr große Druckhöhe von etwa 450 bis 500 mm Wassersäule aufweist.
6. Windsichter nach Anspruch 1, **dadurch gekennzeichnet**, daß die untere Auslaßschwelle (30) rückwärts angeordnet und verstellbar ist, sodaß die durch das Band strömende Luft
- gegenüber der entlang des Bandes strömenden Luft überwiegt.
7. Windsichter nach Anspruch 1, **dadurch gekennzeichnet**, daß der Sichter durch eine hin- und hergehende Bewegung niederer Frequenz im Bereich von 150 bis 450 Schwingungen pro Minute und einem Amplitudenbereich zwischen 10 bis 40 mm angetrieben sein kann.



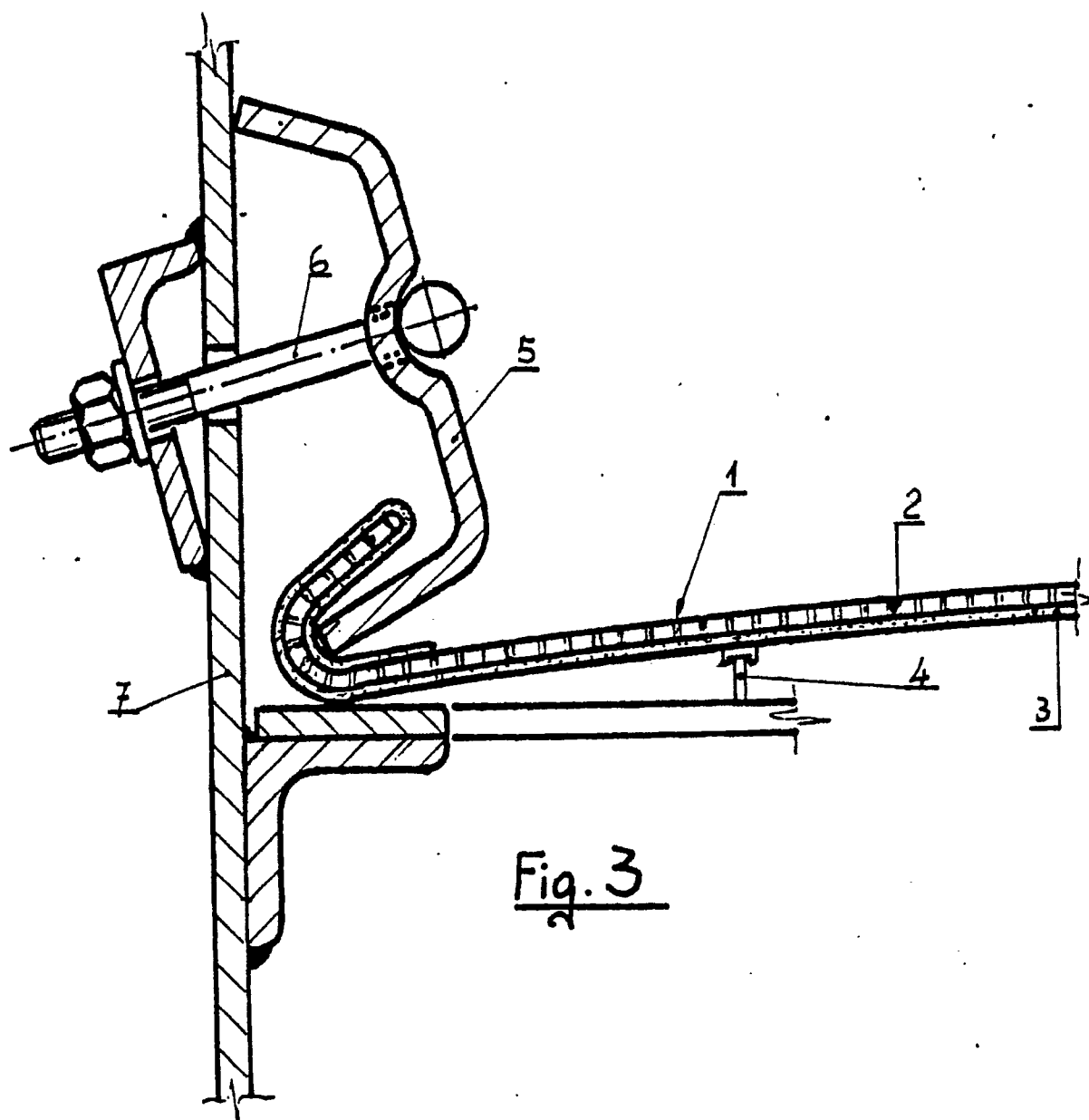
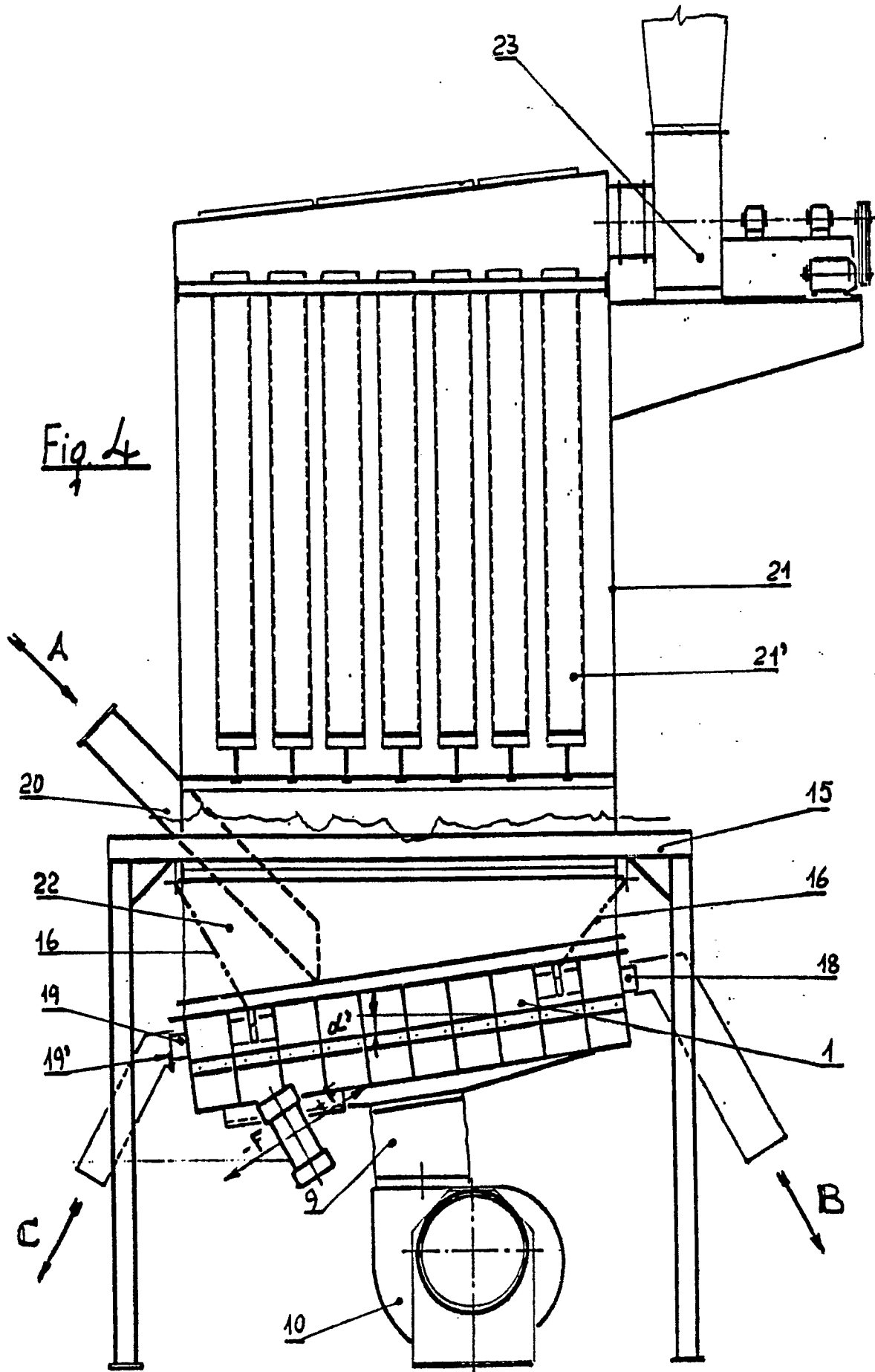
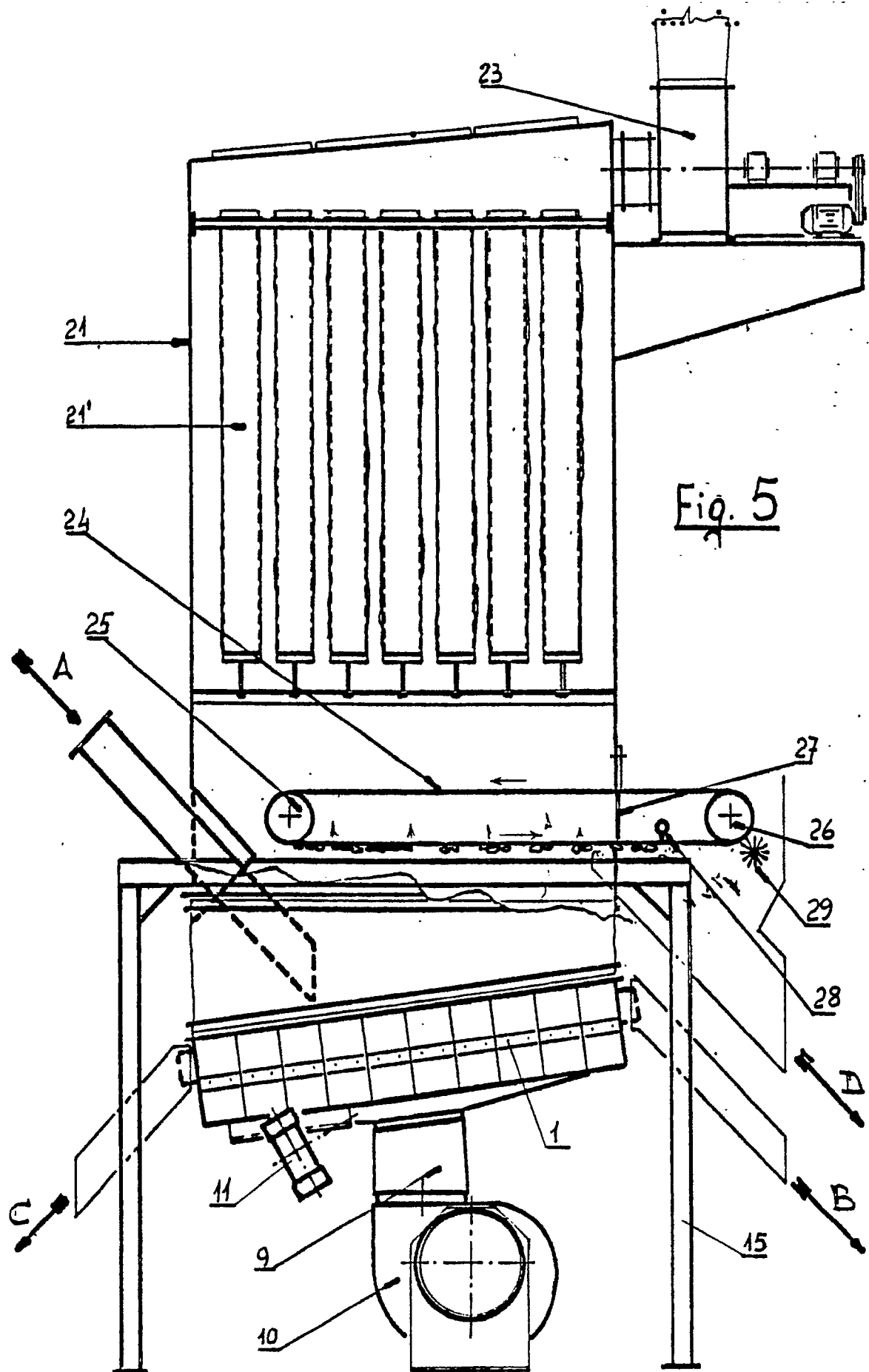


Fig. 3





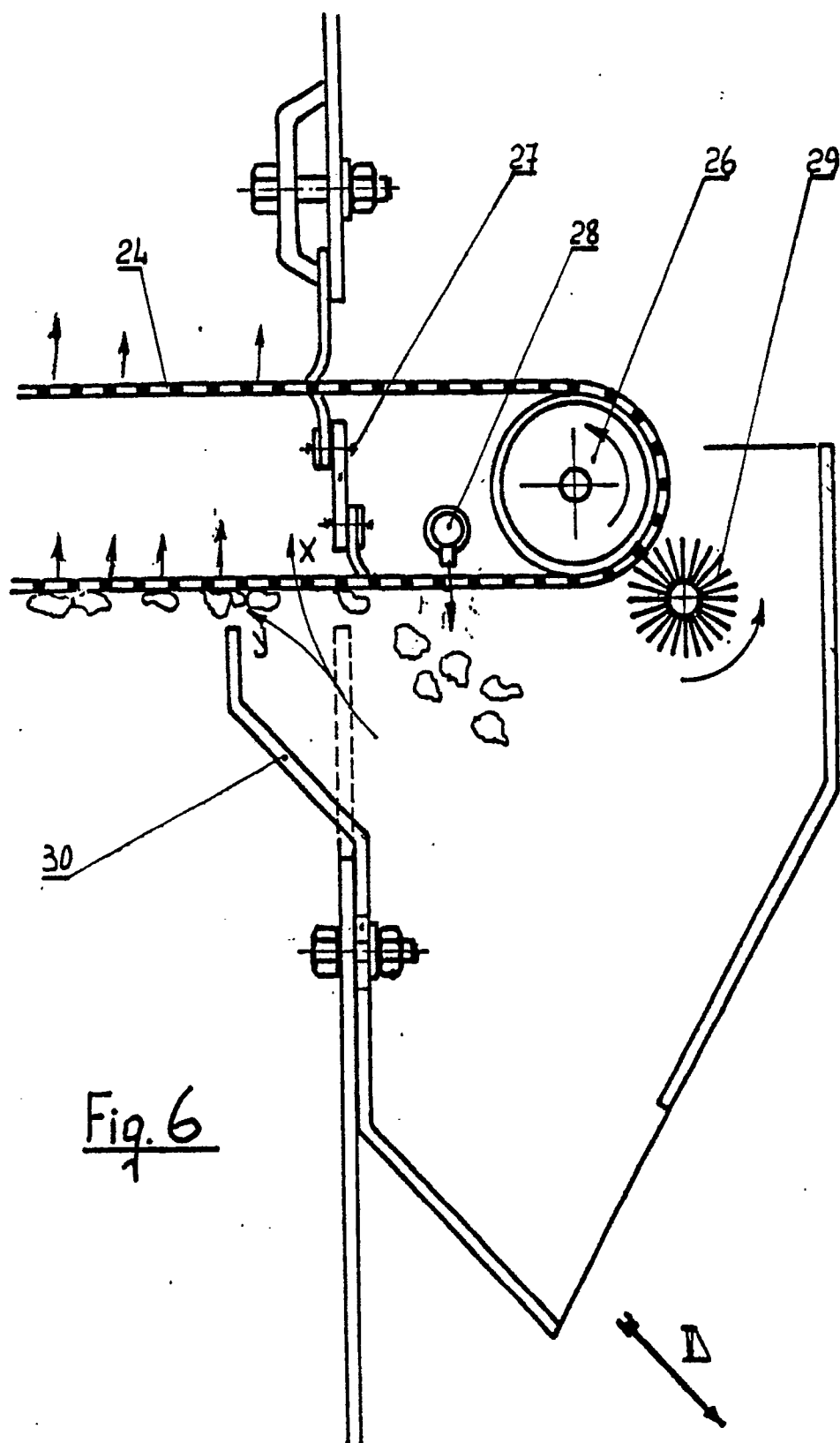


Fig. 6