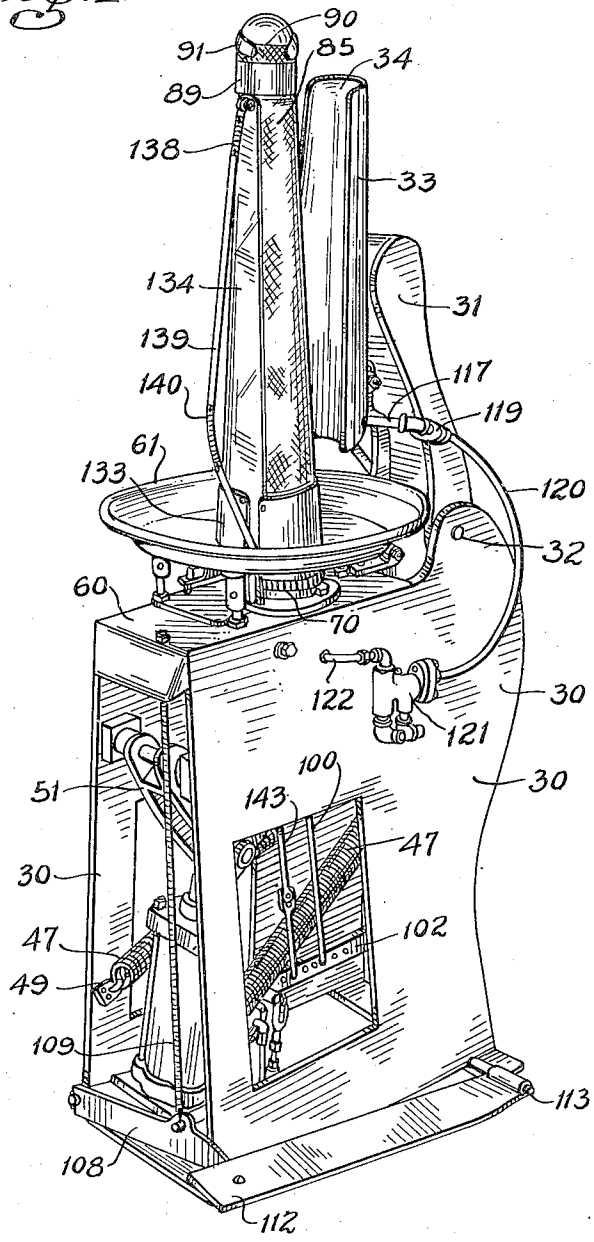
Jan. 24, 1950 E. I. PEARSON 2,495,281

SLEEVE PRESSING MACHINE

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Fig. 1.



INVENTOR.

Edwin I. Pearson

BY

Carlos G. Stratton

ATTORNEY.

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SLEEVE PRESSING MACHINE

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Fig. 2.

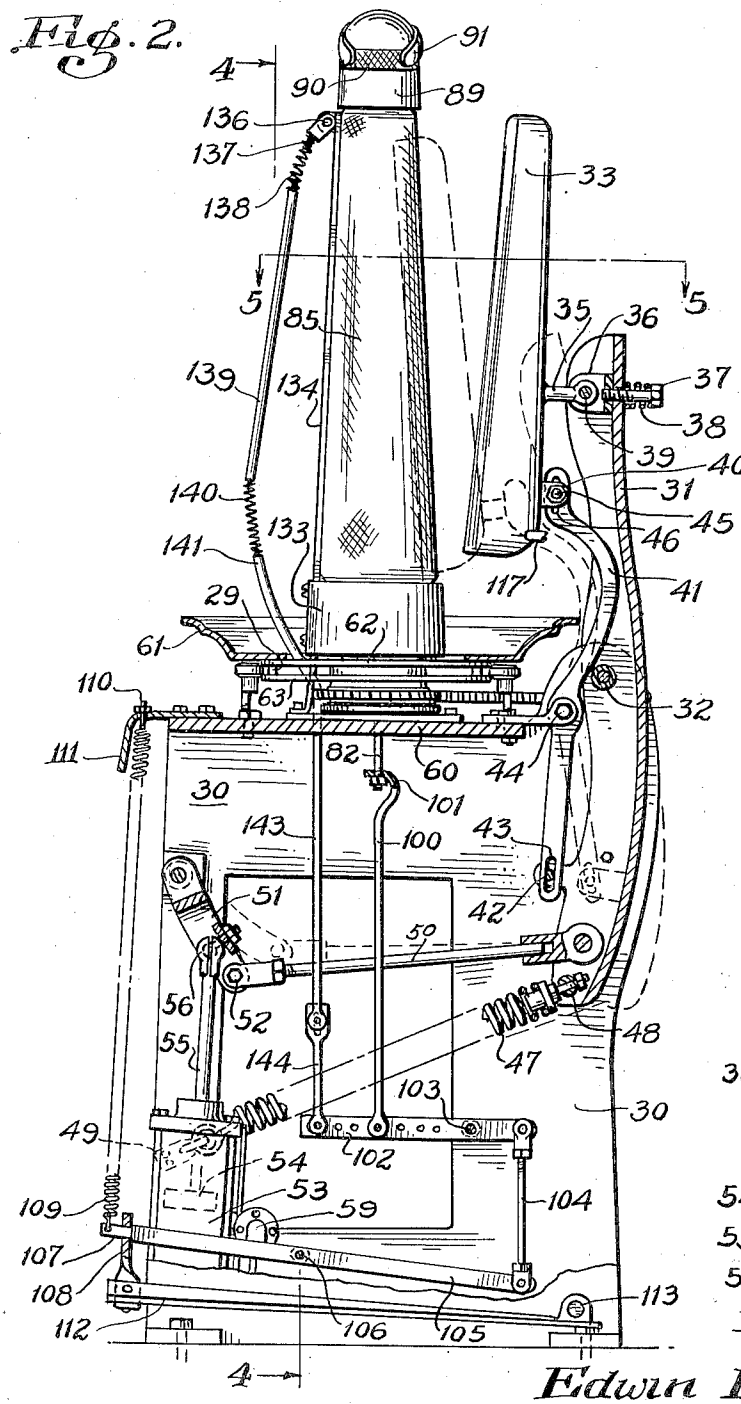
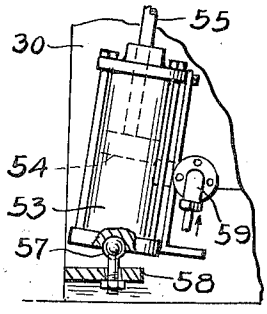


Fig. 3.



INVENTOR.

Edwin I. Pearson

BY

Carlos G. Stratton

ATTORNEY.

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SLEEVE PRESSING MACHINE

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Fig. 4.

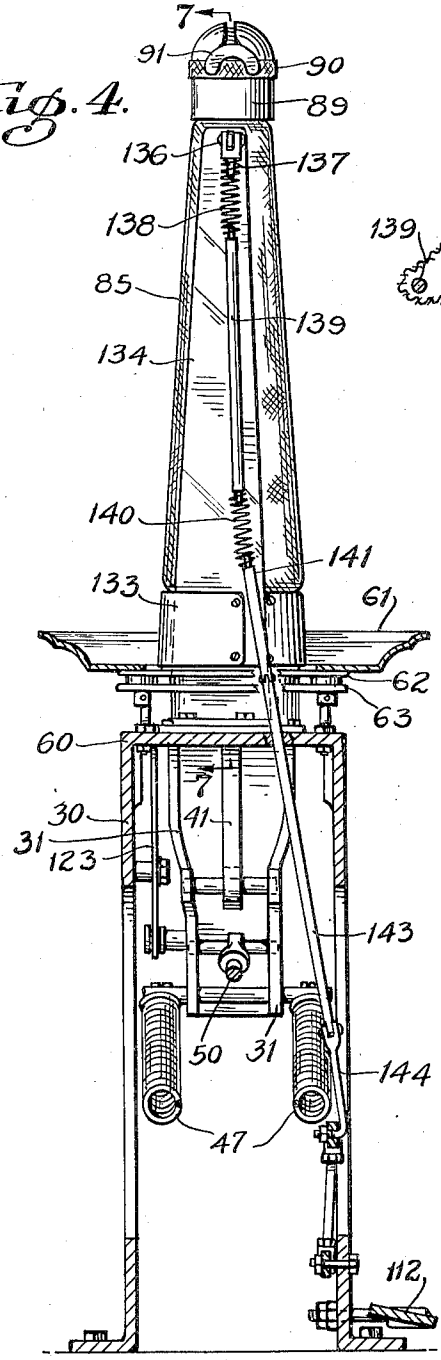


Fig. 5.

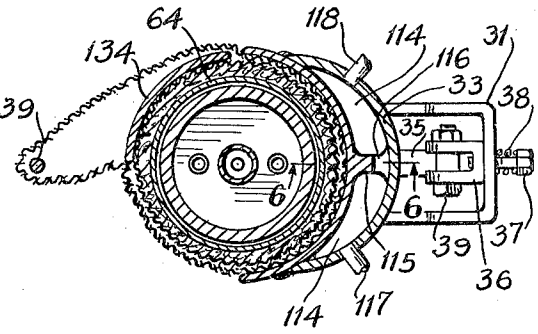
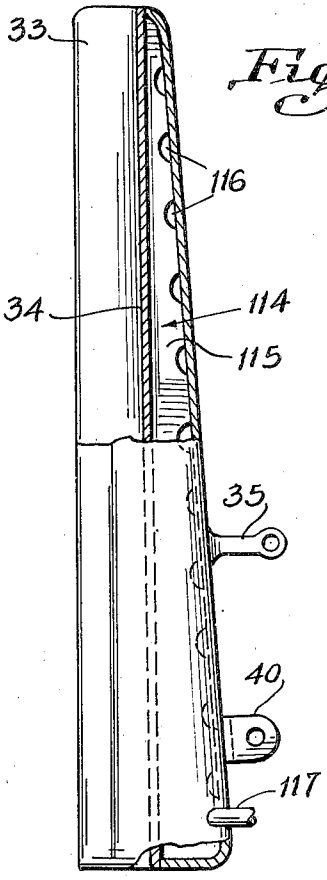


Fig. 6.



INVENTOR.

Edwin I. Pearson

BY Carlos G. Stratton

ATTORNEY.

Jan. 24, 1950 E. I. PEARSON 2,495,281

SLEEVE PRESSING MACHINE

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Fig. 8.

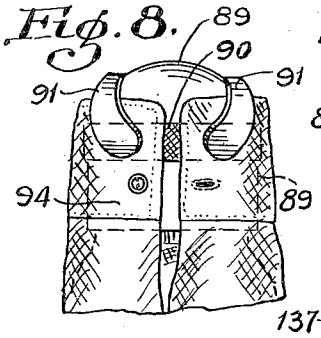


Fig. 7.

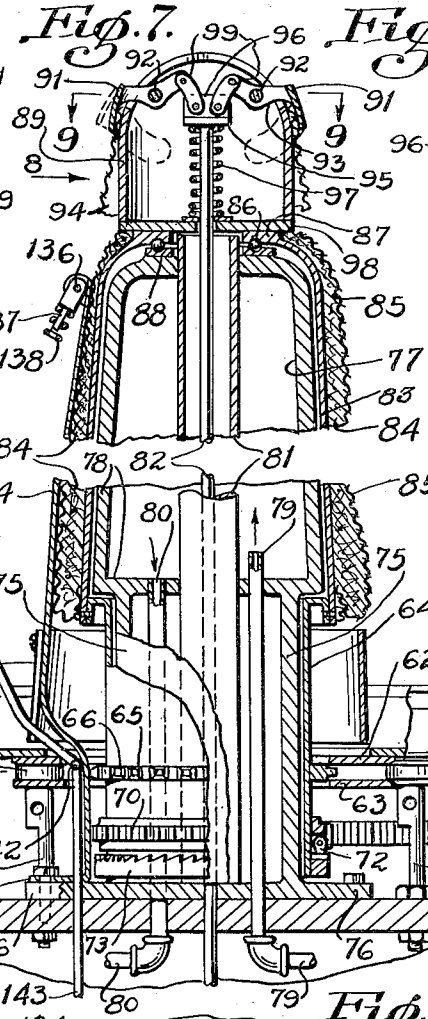


Fig. 9.

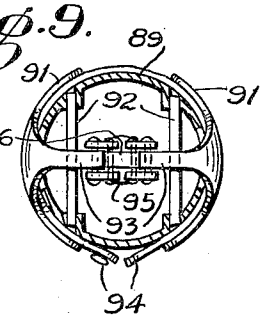


Fig. 10.

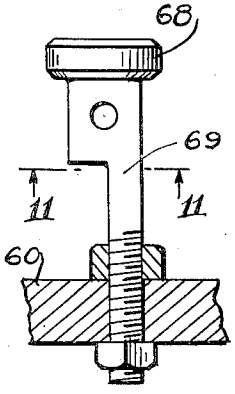


Fig. 12.

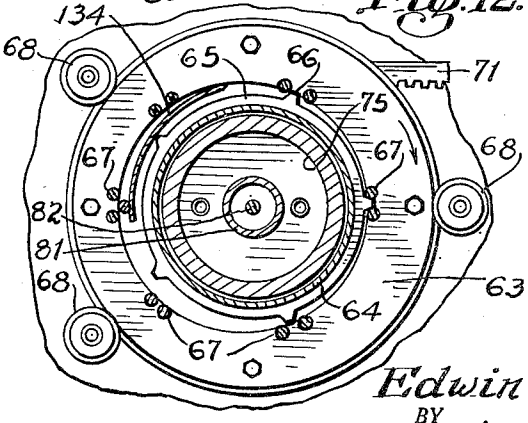
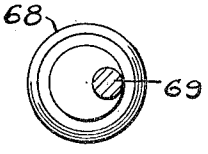


Fig. 11.



INVENTOR.
Edwin I. Pearson
BY
Carlos G. Stratton
ATTORNEY.

Jan. 24, 1950 E. I. PEARSON 2,495,281

SLEEVE PRESSING MACHINE

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Fig. 13.

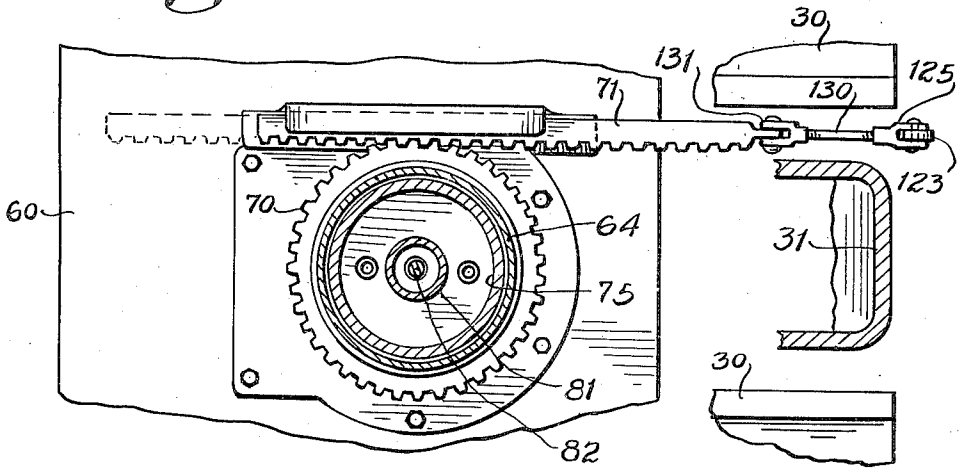
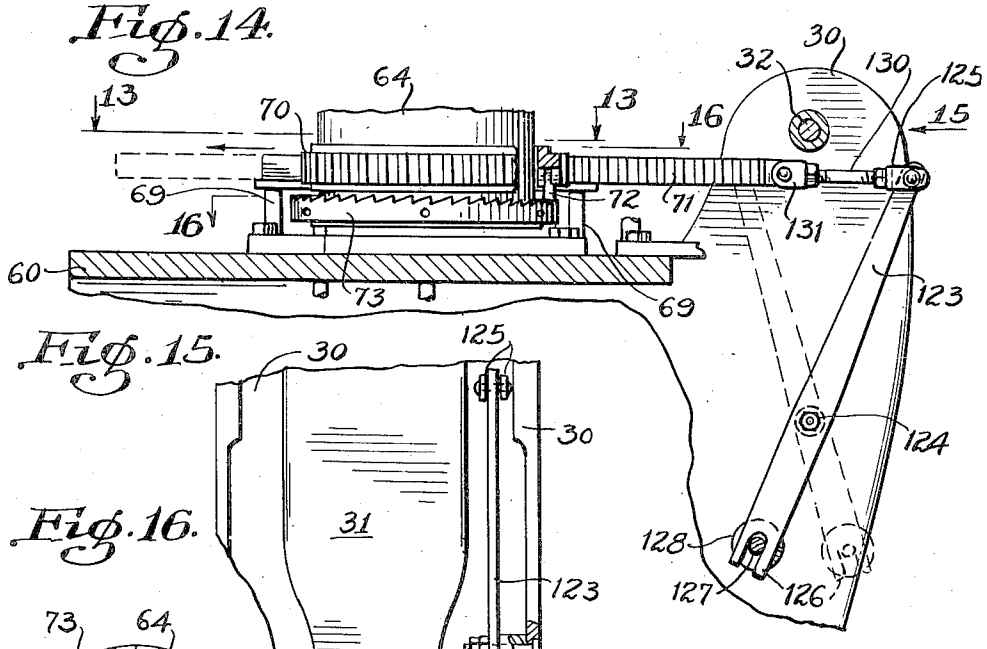


Fig. 14.



INVENTOR.
Edwin I. Pearson
BY
Carlos G. Stratton
ATTORNEY.

Patented Jan. 24, 1950 2,495,281

UNITED STATES PATENT OFFICE

2,495,281

SLEEVE PRESSING MACHINE

Edwin I. Pearson, Salt Lake City, Utah, assignor to Louis N. Strike, Salt Lake City, Utah

Application January 2, 1945, Serial No. 571,035

32 Claims. (Cl. 223—57)

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My invention relates to a sleeve pressing machine for laundries, which is particularly adapted for pressing the sleeves of laundered shirts.

An object of the invention is to provide means for automatically holding sleeves of different sizes against a buck for pressing, and to provide such means in yielding form.

Still other objects are to provide a rotary buck upon which sleeves can be pressed and to provide means for peeling the shirt sleeves from the buck after pressing, during the rotation thereof.

A further object is to provide mechanism in such a pressing machine to prevent the cuff of the shirt from becoming twisted during the pressing operation, and as an exemplification of such object, to provide a separately rotatable cuff support which follows rotation of the rotary buck but is capable of independent rotation, to avoid twisting of the cuff.

A still further object is to provide a generally upright, rotary mandrel for pressing a shirt sleeve, to facilitate drawing a sleeve thereon and removing a sleeve therefrom, in combination with a rotating pan for supporting the rest of the shirt while the sleeve is being pressed.

My invention also has for its objects to provide such means that are positive in operation, convenient in use, easily installed in a working position and easily disconnected therefrom, economical of manufacture, relatively simple, and of general superiority and serviceability.

The invention also comprises novel details of construction and novel combinations and arrangements of parts, which will more fully appear in the course of the following description. However, the drawings merely show and the following description merely describes embodiments of the present invention, which are given by way of illustration or example only.

In the drawings, like reference characters designate similar parts in the several views.

Fig. 1 is a perspective view of an embodiment of my invention.

Fig. 2 is a vertical section through my present pressing machine, showing parts thereof in elevation.

Fig. 3 is a broken detail elevation, partly in section, showing the support for a prime mover provided for my pressing mechanism.

Fig. 4 is a section taken on the offset line 4—4 of Fig. 2.

Fig. 5 is an enlarged transverse section, taken on the line 5—5 of Fig. 2, showing the head in a pressing machine.

Fig. 6 is an enlarged elevation, partly in section, of a pressing head comprised in the invention, taken on the line 6—6 of Fig. 5.

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Fig. 7 is an enlarged broken section, taken on the offset line 7—7 of Fig. 4.

Fig. 8 is a broken elevational view, looking in the direction of the arrow 8 of Fig. 7.

Fig. 9 is a horizontal section taken on the line 9—9 of Fig. 7.

Fig. 10 is an enlarged elevation, partly in section, of a detail hereinafter more fully described.

Fig. 11 is a transverse section taken on the line 11—11 of Fig. 10.

Fig. 12 is a horizontal section taken on the line 12—12 of Fig. 7.

Fig. 13 is a horizontal section taken on the line 13—13 of Fig. 14, showing driving mechanism for my pressing machine.

Fig. 14 is a broken, vertical section of said driving mechanism.

Fig. 15 is a broken elevation, partly in section, looking in the direction of the arrow 15 of Fig. 14.

Fig. 16 is a section taken on the offset line 16—16 of Fig. 14.

Referring more in detail to the drawings, reference numbers 30 indicate side plates of my pressing machine, which pivotally support a generally upright lever 31, which is shown pivoted on the side plates at 32. The lever 31 carries a pressing head 33 which has a concave face 34 shaped to fit the side of a truncated cone. The pressing head is mounted on the lever 31 by means of an arm 35 projecting rearward from the head 33, which arm is pivotally connected to a clevis 36 bolted within the lever 31, as shown at 37. The bolt 37 is free to slide in an aperture in the lever 31 but a spring 38 coiled around the shank of the bolt 37 tends to hold the head 33 yieldingly upon the lever. A pin 39 fastens the arm 35 within the clevis 36.

The head 33 also has a rearwardly projecting lug 40 to which is pivoted another lever 41. The two levers 31 and 41 are pivotally connected to each other by a pin and slot arrangement shown at 42 and 43, in order to compound same. The lever 41 is fulcrumed at 44 on the frame of the machine. A pin and slot arrangement 45 and 46 is also provided by the lug 40 and the adjacent arm of the lever 41.

The compound leverage of the two levers 31 and 41 is arranged to advance the pressing head 33 forward to a pressing position at the proper angle.

Spring means 47 tends to move the head 33 away from a pressing position by reason of a connection of the spring with the pivot mounting

48 upon the lever 31 and a relatively fixed lug 49 on a side plate 30 of the machine.

The mechanism to move the head to its pressing position through the intermediary of the levers comprises a toggle that has pivoted links 50 and 51 that are connected together at 52. A prime mover consists of a cylinder 53 and its contained piston 54. The piston rod 55 has a ball and socket connection 56 with said toggle, whereby the head is moved to a pressing position when the toggle is at least partially straightened out, as shown in broken lines in Fig. 2.

The cylinder 53 has a ball and socket connection 57 with a platform 58 mounted between the side plates 30, whereby the cylinder 53 may follow the swinging movements of the toggle links. An inlet fitting 59 supplies pressure fluid, such as compressed air, to the cylinder 53, from a source (not shown).

A table 60, disposed upon the side plates 30 and bridging the space therebetween, supports a rotating pan 61. Spaced annuli 62 and 63 rotatably support the pan 61 about a rotating column 64. It is believed clear without further illustration that the pan 61 may be fastened to the upper annulus 62 by any suitable means, and that the annuli may be fastened together and spaced apart by suitable tie and spacer means.

A toothed ring 65 is arranged between the annuli 62 and 63, the teeth being shown at 66. The ring 65 is mounted upon the column 64. The pins 67 are disposed in pairs between and arranged around the inner opening provided by the annuli. It will be noted that the column 64 is eccentric with regard to the annuli 62, 63 for a purpose hereinafter described.

Rollers 68 ride between the annuli 62 63 to rotatably support the pan 61. Eccentric bolts 69 support the rollers upon the table 60. The eccentric arrangement permits in-and-out adjustment of the rollers 68 with respect to the periphery of the annuli.

As the toothed ring 65 rotates, its teeth engage the pins 67 as suggested in Fig. 12, and effect conjoint rotation of the pan 61 and column 64.

Mechanism to rotate the column 64 comprises a ring gear 70 that is engaged and driven by a reciprocating rack 71. A pivoted dog 72 on the ring gear 70 interlocks with a crown gear 73 when the ring gear 70 is rotated in one direction, and idly slides along the crown gear when the ring gear is rotated in the opposite direction, whereby to drive the column in one direction by reciprocation of the rack. The crown gear 73 is pinned to the column 64 as suggested at 74 in Fig. 16.

A fixed housing 75, around which the column 64 rotates, is mounted upon the table 60 by flange means 76. Integral with the housing 75 and arranged thereon is a steam chest 77 in the shape of a truncated cone. A partition 78 divides the interior of the housing 75 from the interior of the steam chest 77. A steam pipe 79 extends through the housing 75 and empties into the steam chest 77, while the pipe 80 discharges the products of condensation from the steam chest. A bushing 81 extends through the steam chest 77 for housing a reciprocating rod 82. The bushing 81 is in steam-tight connection with the chest 77 and stationary therewith.

Mounted to rotate around the steam chest is a mandrel 83 that is fastened upon the rotating column 64. Suitable padding 84 is applied to the mandrel 83 and a fabric covering 85 is placed upon the padding. The mandrel 83 is supported upon ball bearings 86 in races between inwardly directed flanges 87 and 88 at the top of the mandrel and the steam chest, respectively.

Independently rotatable upon the top of the mandrel 83 is a cuff support 89. It will be noted in Fig. 7 that there is no positive connection between the cuff support and the mandrel top flange 87. There is some friction between those parts which restricts the independent rotation of the cuff support 89. However, the cuff support is sufficiently rotatable independently to prevent the cuff of a shirt sleeve from being twisted with respect to the rest of the sleeve.

Clamp means are provided for holding a cuff against the knurled portion 90 on the cuff support 89. The clamp means comprises clamping members 91 arranged to clamp a cuff 94 in place on the support 89.

The members 91 are mounted upon bell cranks 93 that are pivoted on shafts 92 mounted inside the cuff support 89.

The reciprocating rod 82 carries thereon a combination abutment 95 and lug 96. A coil spring 97 abuts against the abutment 95 and the floor 98 of the cuff support, tending to raise the rod 82. The reactive force of spring 97 also effects frictional engagement of said cuff support and mandrel 83. Links 99 pivotally connect the lug 96 with the bell cranks 93, so that when the rod 82 is raised by the spring, the clamp members 91 swing to cuff clamping positions.

Operating means for moving the rod 82 downwardly against the action of the spring 97 comprises a bar 100 having a hooked end 101 that engages the rod 82. The bar 100 is connected with a lever 102 which is fulcrumed on a shaft 103 between the side plates 30 of the machine. An adjustable link 104 connects the lever 102 with another lever 105, which in turn is fulcrumed on the shaft 106 also between the side plates 30. The lever 105 has a reduced end 107 that projects through an opening in a pivoted bar 108 at the front of the machine. The coil spring 109 connects the projecting end 107 with a relatively fixed bolt 110 on a front plate 111 that is supported on the table 60. The spring 109 tends to assist the spring 97 through said system of leverage. The assistance is rather in nature of spring 109 supporting the weight of said system of levers so that said weight normally cannot act to depress rod 82 against the lifting force of spring 97. A pedal 112 pivoted at 113 near the rear of the machine, is mounted on a bar to depress both springs when an operator steps on the pedal and thereby raise cuff clamps 91.

The pressing head 33 contains a steam chamber 114. Bracing wall 115, shown in Figs. 5 and 6, may be employed for strengthening same. Passages 116 through the wall 115 connect the halves of the chamber 114 with each other. Pipes 117 and 118 supply steam to the chamber portions 114 at the lower ends thereof. Steam fitting means 119 are shown connected to the pipe 117 on one side of the pressing head 33. One end of a flexible conduit 120 is connected to a steam fitting 119, and the other end of said conduit is connected to a steam fitting 121, that is provided with a pipe 122 onto which is fitted a flexible conduit, not shown, which leads to and connects with the pipe 118 on the other side of the pressing head.

Actuating means for the rack 71 consists of a lever 123 pivoted on a side plate 30, at 124. The lever 123 is connected at one end with a clevis

125 projecting from the end of the rack 71, and the other end of the lever is bifurcated as shown at 126, to straddle a pin 127 carrying spaced collars 128. The inner end of the pin 127 is threaded to receive nuts 129 for locking the pin on the main lever 31. Thus swinging movement of the lever 31 will cause the smaller operating lever 123 to swing from the full line position to the broken line position in Fig. 14, about the fixed point 124. This swinging movement of the lever 123 is transmitted into reciprocating movement of the rack 71. The clevis 125 is screwed upon a bar 130 that in turn has a clevis connection 131 with the rack 71. This arrangement permits a slight upward and downward oscillation of the bar 130 necessary to accommodate the swinging movements of the lever 123.

A stationary collar 133 is arranged around the column 64, within the pan 61. The collar is bolted to a lower portion of a relatively stationary stripper or peeling plate 134, which strips or peels the sleeve from the rotating buck after pressing. The stationary plate 134 extends down inside the collar 133 and has a foot 135 that is bolted to a flange 76 at the bottom of the housing 75, by means of one of the bolts 69, as perhaps best shown in Fig. 7. The leading edge of the stripper or peeling plate 134 is arranged in close proximity to the rotating buck 85 to facilitate this stripping or peeling action.

Sleeve distending means are provided at one side of the pressing buck in order to effectively smooth the sleeve at the opposite side of the buck, opposite the pressing head. Pivotally connected on a pin 136 at the top of the stripper or peeling plate 134 is a wire 137 that extends into and is suitably connected to one end of a coil spring 138 by any suitable means, such as by welding.

A bar or tubular member 139 has endwise projecting pins suitably fastened in the opposite end of the spring 138 and to an end of a spring 140, respectively. The other end of the spring 140 is similarly fastened to a rod 141 pivoted at 142 to a pull bar 143. The pull bar 143 is in turn pivotally connected with an arm 144 adjustably connected to the lever 102 and actuated by the pedal 112 in the same manner as the bar 100 hereinbefore described.

The springs 138 and 140 normally tend to swing the sleeve distender outward from the buck 85 so that a sleeve thereon is smoothed out on the opposite side of the buck from the distender.

In the operation of my present sleeve press, a sleeve is passed over the sleeve buck 85 and the remainder of the shirt laid in the pan 61. The pan 61 rotates with the buck to prevent twisting of the shirt sleeve on the shirt.

The operator steps on the pedal 112, which draws the extender member 139 toward the buck 85 and simultaneously moves the cuff clamps away from the cuff mandrel 89. The shirt sleeve may then be drawn down over the buck 85, over the extender member, and over the cuff mandrel. When the cuff has been smoothed out around its mandrel 89, the pedal 112 may be released, which causes the spring 97 to urge the clamp 91 to clamping positions with respect to the cuff 94. Releasing the pedal 112 also causes the extender, under influence of the spring 109, to push the sleeve outward away from the buck on the side opposite the pressing side. The springs 138 and 140 tend to smooth the shirt sleeve on the pressing side, opposite the pressing head 33, after the pedal 112 has been released.

It is believed clear that while the cuff mandrel 89 substantially follows rotary movements of the buck 85 due to the friction of their abutting faces, there is no positive connection that compels this conjunctive movement. For this reason any tendency of the cuff to become twisted upon the sleeve is counteracted, for the tendency of the cuff to so twist will cause the mandrel 89 to slip with respect to the buck 85, until the tendency to twist is overcome. It will be noticed that there is a rotary or swivel connection between the rod 82 and the hooked end 101 to permit rotation of the rod 82 with the cuff mandrel 89.

The springs 138, 140 have the further distinct advantage of fitting large sleeves or small ones, thus readily adapting the present press to different sizes of shirts. Moreover these springs provide rounded corners in the sleeve.

It is believed clear without further illustration that any suitable means, manual or pedal, may be employed for controlling the prime mover 53 and the pressure fluid for operating it. Suitable means are shown in United States Letters Patent No. 2,332,830, issued October 26, 1943, to Otto H. Pearson.

Suffice to say, when the pressure fluid is admitted to the cylinder 53 to actuate the piston 54 therein, the piston rod moves the toggle links 50 and 51 towards a straightened relationship, such as the broken line position in Fig. 2. This movement swings the lever 31 about its pivot, as also indicated by broken lines in said Fig. 2. The pressing head 33 is thereby bodily moved forward to a position of engagement with a shirt sleeve on the buck 85, as also indicated in broken lines in Fig. 2. The compound action of the levers 31 and 41 moves the pressing head 33 forward to a position substantially parallel with the buck. This action aids in moving the lower end as well as the upper end of the head 33 away from the buck between pressing operations, so that the pressed shirt may be removed and a new one placed on the buck.

Due to the mechanism perhaps best shown in Figs. 13 to 16, each time the pressing head is moved rearward away from the buck 85, the rack 71 rotates the buck 85, automatically thus presenting a different portion of the sleeve to the pressing head for the next pressing operation. Moreover as the buck rotates, the relatively stationary plate 134 acts as a peeler or stripper to remove the pressed portion of the shirt sleeve from the buck since there is a tendency for the pressed portion of the sleeve to stick on the buck after pressing. This peeling or stripping is done automatically by rotation of the buck. The pan 61, as stated, rotates with the buck 85.

It is believed clear that the relatively large springs 47 return the pressing head 33 to its inoperative position when the press is released from the piston 54. During pressing, the pressing head 33 is self-adjusted to the buck by means of the pivotal connection 39 and the spring 38. This is of distinct advantage, since there may be unevenness in the padding on the buck and then, of course, there is a different relationship between the head and buck when the padding is first applied to the buck and after it has become matted down. This is compensated for by said means.

It will be noted, especially in Fig. 12, that the vertical axis of the buck is eccentric with respect to the vertical axis of the driving plate 63. This is necessitated by the fact that the stationary peeler or stripper plate 134 extends up through

the rotating pan 61 and it is desired to have a minimum of space between the rotating pan and adjacent the stationary peeler through which parts of the shirt being pressed can get into the working mechanism.

While I have illustrated and described what I now regard as the preferred embodiments of my invention, the constructions are, of course, subject to modifications without departing from the spirit and scope of my invention. I, therefore, do not wish to restrict myself to the particular forms of construction illustrated and described, but desire to avail myself of all modifications that may fall within the scope of the appended claims.

Having thus described my invention, what I claim and desire to secure by Letters Patent is:

1. In a sleeve pressing machine, a buck rotatable about an axis within the body of the buck, a pressing head movable to and from a position in engagement with the buck in different rotary positions of the buck, and means to rotate the buck upon movement of the pressing head.

2. In a pressing machine, a rotary buck, a relatively stationary peeling means arranged alongside the buck to peel pressed material from the buck during rotation thereof, and a pressing head movable to and from a position in engagement with the buck, said position being around the periphery of the buck from the peeling means.

3. In a sleeve pressing machine, a rotary buck, a pressing head movable to and from a position in engagement with the buck in different rotary positions of the buck, and resilient means arranged longitudinally of the buck to yieldingly hold a side of a sleeve away from the buck to smooth the sleeve in the area opposite the pressing head.

4. In a sleeve pressing machine, a rotary buck, a pressing head movable to and from a position in engagement with the buck in different rotary positions of the buck, and substantially non-rotary resilient means arranged to yieldingly hold a side of a sleeve away from the buck at a point in the periphery of the buck other than the portion opposite the pressing head.

5. In a sleeve pressing machine, a rotary buck, a relatively stationary peeling means arranged alongside the buck to peel pressed material from the buck during rotation thereof, resilient means supported on the stationary peeling means and arranged to yieldingly hold a side of a sleeve away from the buck, and a pressing head movable to and from a position in engagement with the buck, said position being around the periphery of the buck from the peeling means and from the resilient means.

6. In a sleeve pressing machine, a rotary buck, a pressing head movable to and from a position in engagement with the buck in different rotary positions of the buck, an element over which a sleeve is adapted to be drawn, movable toward and from the buck at a side of the buck other than at the side opposite the pressing head, resilient means tending to move said element away from the buck, and operator-controlled means connected to move said element toward the buck against the action of the resilient means.

7. In a sleeve pressing machine, a rotary buck, mechanism connected at an end of the buck to rotate same, the opposite end of the buck being unsupported to draw a sleeve thereover, sleeve-extending means having a relatively fixed support adjacent the unsupported end of the buck and extending first outwardly alongside the buck and then inwardly toward the supported end thereof, and a pressing head movable to and from a position in engagement with the buck, said position being around the periphery of the buck from the sleeve-extending means.

8. In a sleeve pressing machine, a rotary mandrel for the body of a sleeve, a cuff support mounted to rotate with the mandrel, the cuff support being capable of rotation independently of the mandrel, to prevent twisting of the cuff with respect to the sleeve, and a pressing head movable to and from a position in which it engages the mandrel in different rotary positions of the mandrel.

9. In a sleeve pressing machine, an upright, rotary mandrel for the body of a sleeve, a cuff support upon the mandrel mounted to rotate with the mandrel, the cuff support being capable of rotation independently of the mandrel but being disposed to frictionally engage and rotate with the mandrel, and a pressing head movable to and from a position in which it engages the mandrel in different rotary positions of the mandrel.

10. In a sleeve pressing machine, a rotary mandrel for the body of a sleeve, a cuff support mounted to rotate with the mandrel, spring means yieldingly tending to effect rotation of the cuff support with the mandrel, the cuff support being capable of rotation independently of the mandrel, against the action of the spring means, and a pressing head movable to and from a position in which it engages the mandrel in different rotary positions of the mandrel.

11. In a sleeve pressing machine, a rotary mandrel for the body of a sleeve, a cuff support mounted to rotate with the mandrel, the cuff support being capable of rotation independently of the mandrel, clamping means operable to clamp a cuff on the support, mechanism yieldingly tending to clamp the clamping means upon the support, said mechanism being connected to be controlled by the operator and extending through the mandrel, and a pressing head movable to and from a position in which it engages the mandrel in different rotary positions of the mandrel.

12. In a sleeve pressing machine, a rotary mandrel for the body of a sleeve, a cuff support mounted to rotate with the mandrel, clamping means operable to clamp a cuff on the support, an element arranged to hold a side of the sleeve away from the mandrel, spring means tending to hold the clamping means against the cuff support, spring means tending to hold the element away from the mandrel, mechanism connected to simultaneously move said clamping means away from the cuff support and the element toward the mandrel, against the action of the spring means, and a pressing head movable to and from a position in engagement with the mandrel, to press a sleeve whose cuff is gripped by the clamping means and whose body is held away from the mandrel to smooth same on the side opposite the pressing head.

13. In a sleeve pressing machine, a rotary pan for receiving the body of the shirt whose sleeve is being pressed, an upright rotary mandrel upon which a sleeve is drawn while the body of the shirt falls by gravity in the pan, mechanism connected to rotate the pan and the mandrel simultaneously, and a pressing head movable to and from a position in engagement with the mandrel.

14. In a sleeve pressing machine, an apertured, rotary pan for the body of the shirt whose sleeve is being pressed, an upright rotary mandrel,

mechanism connected to drive the mandrel through the aperture in the pan, a relatively stationary peeling means alongside the mandrel disposed in the aperture in the pan, and a relatively stationary collar arranged to prevent a shirt in the pan from engaging the driving mechanism for the mandrel.

15. In a sleeve pressing machine, an apertured rotary pan for the body of the shirt whose sleeve is being pressed, an upright rotary mandrel eccentrically arranged with respect to the aperture in the pan, a peeling means alongside the mandrel disposed on the side of the mandrel where the greater distance exists between the eccentric mandrel and the edge of the aperture in the pan, and mechanism connected to drive the mandrel through the aperture in the pan and connected to rotate the therewith eccentrically arranged pan.

16. In a sleeve pressing machine, an apertured, rotary pan for the body of the shirt whose sleeve is being pressed, an upright, rotary mandrel eccentrically arranged with respect to the aperture in the pan, a peeling means alongside the mandrel disposed on the side of the mandrel where the greater distance exists between the eccentric mandrel and the edge of the aperture in the pan, mechanism connected to rotate the mandrel intermittently through the aperture in the pan, and means to transmit the intermittent rotation of the mandrel to the pan.

17. In a sleeve pressing machine, a rotary mandrel, a pressing head movable to and from a position in engagement with the mandrel, means connected to transmit movement of the pressing head into rotary movement of the mandrel between pressing engagements of the head with the mandrel, and operator-controlled mechanism connected to operate the pressing head at the instance of the operator.

18. In a sleeve pressing machine, a rotary mandrel, a peeling means alongside the mandrel, a pressing head movable to and from a position in engagement with the mandrel, means connected to transmit movement of the pressing head away from the mandrel into rotary movement of the mandrel, effecting operation of the peeling means and presenting a new mandrel face to the pressing head, and operator-controlled mechanism connected to operate the pressing head at the instance of the operator.

19. In a sleeve pressing machine, a rotary mandrel, a pressing head movable to and from a position in engagement with the mandrel, rack and ratchet means connected to transmit movement of the pressing head into rotary movement of the mandrel between pressing engagements of the head with the mandrel, and operator-controlled mechanism connected to operate the pressing head at the instance of the operator.

20. In a pressing machine, a rotary mandrel having a pressing surface thereon, a relatively stationary steam chest within the mandrel disposed to transmit heat thereto, connecting means to supply steam to the chest and to deliver products of condensation therefrom, and a pressing head movable to and from a position in which it engages the mandrel in different rotary positions of the mandrel.

21. In a pressing machine, a rotary mandrel having a hollow steam chest therein for heating the mandrel, and a pressing head movable to and from a position in which it engages the mandrel in different rotary positions of the mandrel.

22. In a sleeve pressing machine, a rotary buck, a pressing head movable to and from a position in engagement with the buck in different rotary positions of the buck, and resilient means arranged longitudinally of the buck at a distance from the side thereof, said means being so formed that it seeks a position away from the buck, and mechanism connected to rotate the buck independently of said resilient means, after pressing material on the buck, whereby the resilient means also acts as a peeling means.

23. In a pressing machine, a rotatable buck, a pressing head movable to and from a position in engagement with the buck, and means arranged on the side of the buck away from the pressing head and associated with the buck for peeling therefrom an article pressed between the pressing head and the buck upon rotation of said buck.

24. In a sleeve pressing machine, a rotatable buck for the body of a sleeve, a cuff support associated with one end of the buck, means to clamp the cuff of a sleeve on said support to thereby hold the body of the sleeve in pressable position with respect to the buck, and a pressing head movable to and from a position in pressing engagement with the buck to press successive portions of the body of the sleeve upon rotation of the buck.

25. In a sleeve pressing machine, a rotatable mandrel for mounting a sleeve, a pressing head, means to move said head to and from a position in pressing engagement with said mandrel to press said sleeve, and means controlled by the movement of said pressing head to rotate said mandrel and thereby press successive portions of said sleeve.

26. In a sleeve pressing machine, a rotatable mandrel for mounting a sleeve, a pressing head, means to move said head to and from a position in pressing engagement with said mandrel to press said sleeve, and means to rotate said mandrel when said head is moved from engagement therewith to bring successive portions of said sleeve into position to be pressed by the head.

27. In a sleeve pressing machine, a rotatable mandrel for mounting a sleeve, a pressing head, means to move said head to and from a position in pressing engagement with said mandrel to press said sleeve, means to rotate said mandrel when said head is moved from engagement therewith to bring successive areas of said sleeve into position to be pressed by the head, and means associated with the mandrel to peel the pressed area of the sleeve from said mandrel during rotation thereof.

28. In a pressing machine having a pressing head member, a buck member, and means for moving one of said members relatively to the other to effect a pressing operation therebetween; resilient and pivoted mounting means yieldingly supporting the pressing head member to effect aligned pressing engagement of said members upon movement of the members together.

29. In a pressing machine having a pressing head member, a buck member and a lever for pivotally moving one of said members relatively to the other to effect a pressing operation therebetween; pivotal and resilient means pivotally and yieldingly mounting the pressing head member on the lever to effect aligned pressing engagement of said members upon movement of the lever in a direction to bring said members together.

30. In a pressing machine having a pressing head and a buck; a pivoted lever arranged to pivotally carry the pressing head toward and from the buck to effect a pressing operation

therebetween, a second lever having a fixed fulcrum and connected at one end to the first lever and actuated thereby, a connection between the other end of the second lever and the pressing head for controlling the pivotal movement of the latter, whereby aligned pressing engagement of the pressing head and buck is effected upon movement thereof together.

31. In a pressing machine having a pressing head and a buck; a pivoted lever arranged to pivotally carry the pressing head toward and from the buck to effect a pressing operation therebetween, a second lever having a fixed fulcrum and connected at one end to the first lever and actuated thereby, a sliding connection between the other end of the second lever and the pressing head for controlling the pivotal movement of the latter with respect to the first lever, whereby, upon movement of the pressing head and buck together, an aligned pressing engagement thereof is effected.

32. In a sleeve pressing machine, a rotatable

buck for mounting a sleeve, a pressing head, means to move said head to and from a position in pressing engagement with the buck to press successive portions of a sleeve on the buck as the latter rotates, and means associated with the buck on the side opposite to the pressing head to tauten the sleeve for substantially creaseless engagement with that portion of the buck that is engaged by the pressing head.

EDWIN I. PEARSON.

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