June 6, 1939. E. J. ABBOTT 2,160,810

WINDING MACHINE

Filed Nov. 24, 1931 7 Sheets-Sheet 1

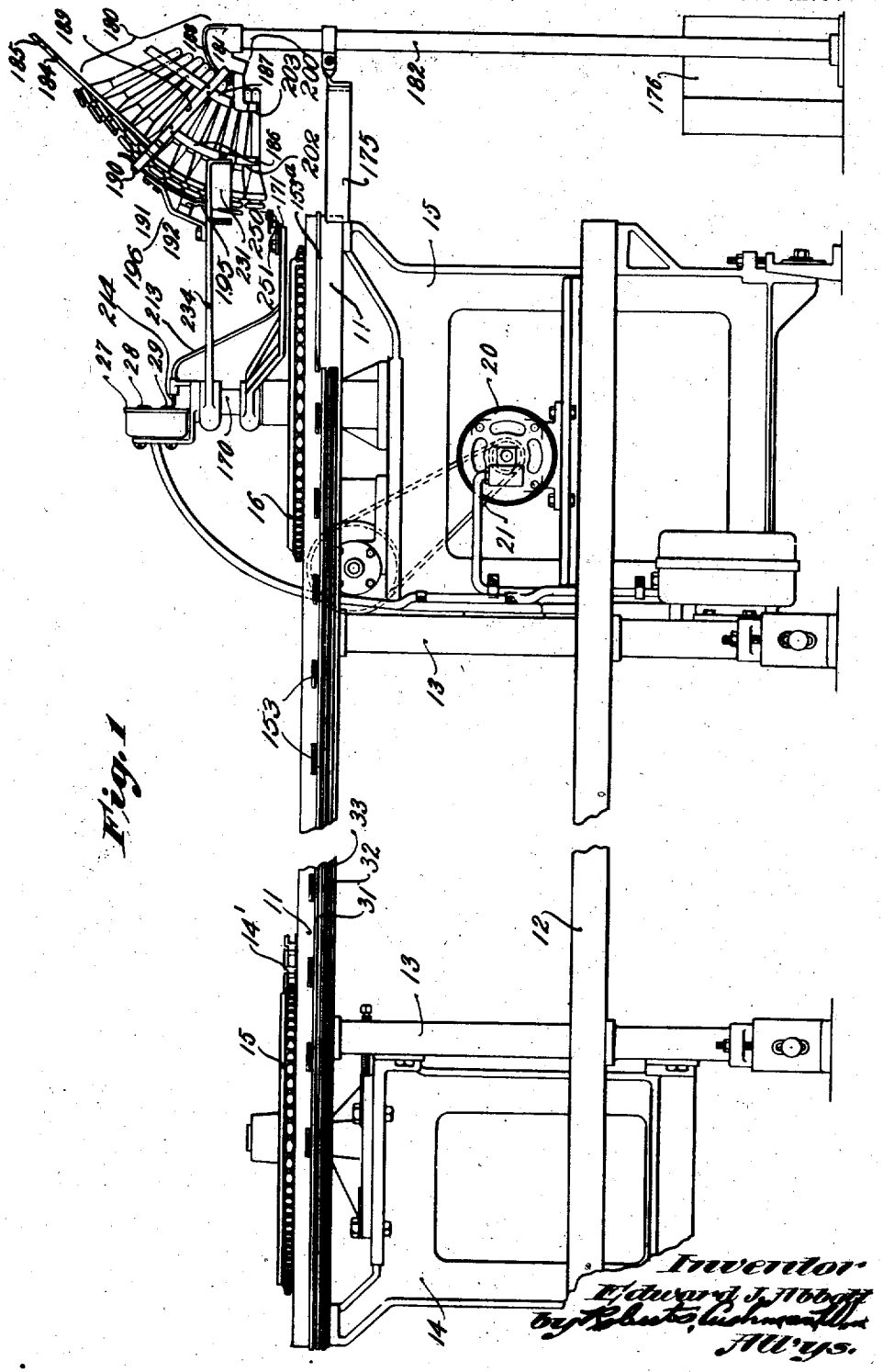


Fig. 1

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WINDING MACHINE

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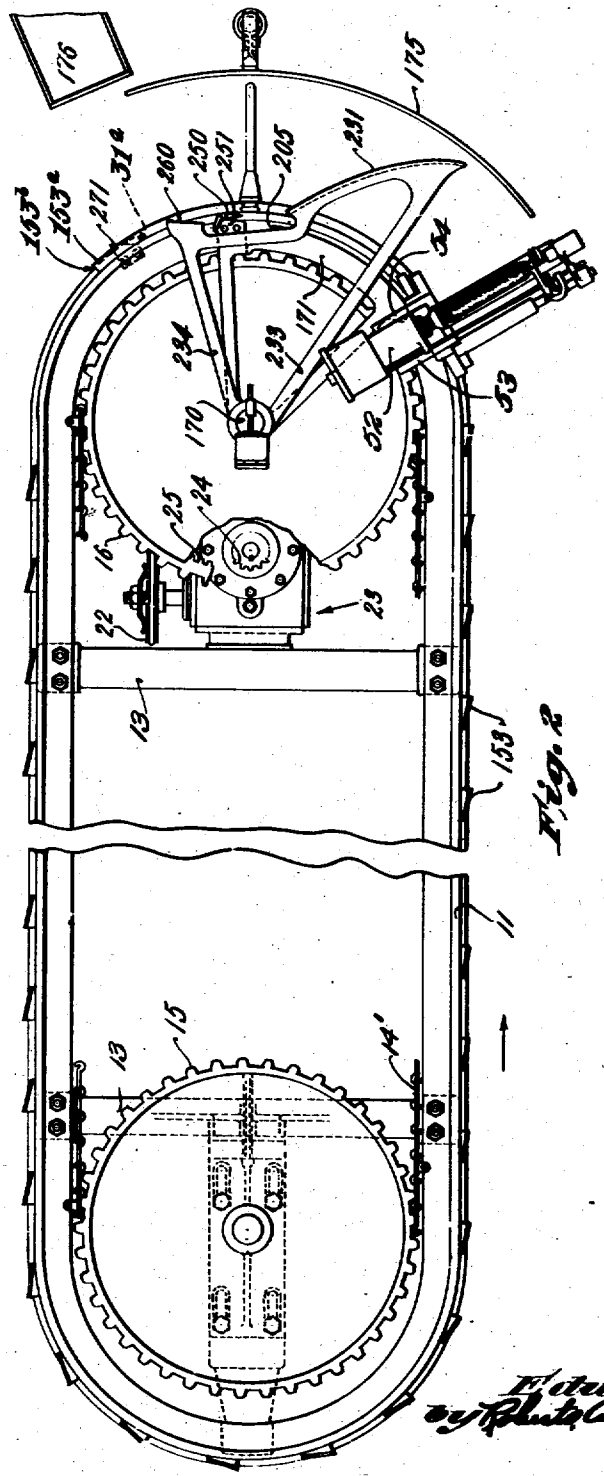


Fig. 2

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WINDING MACHINE

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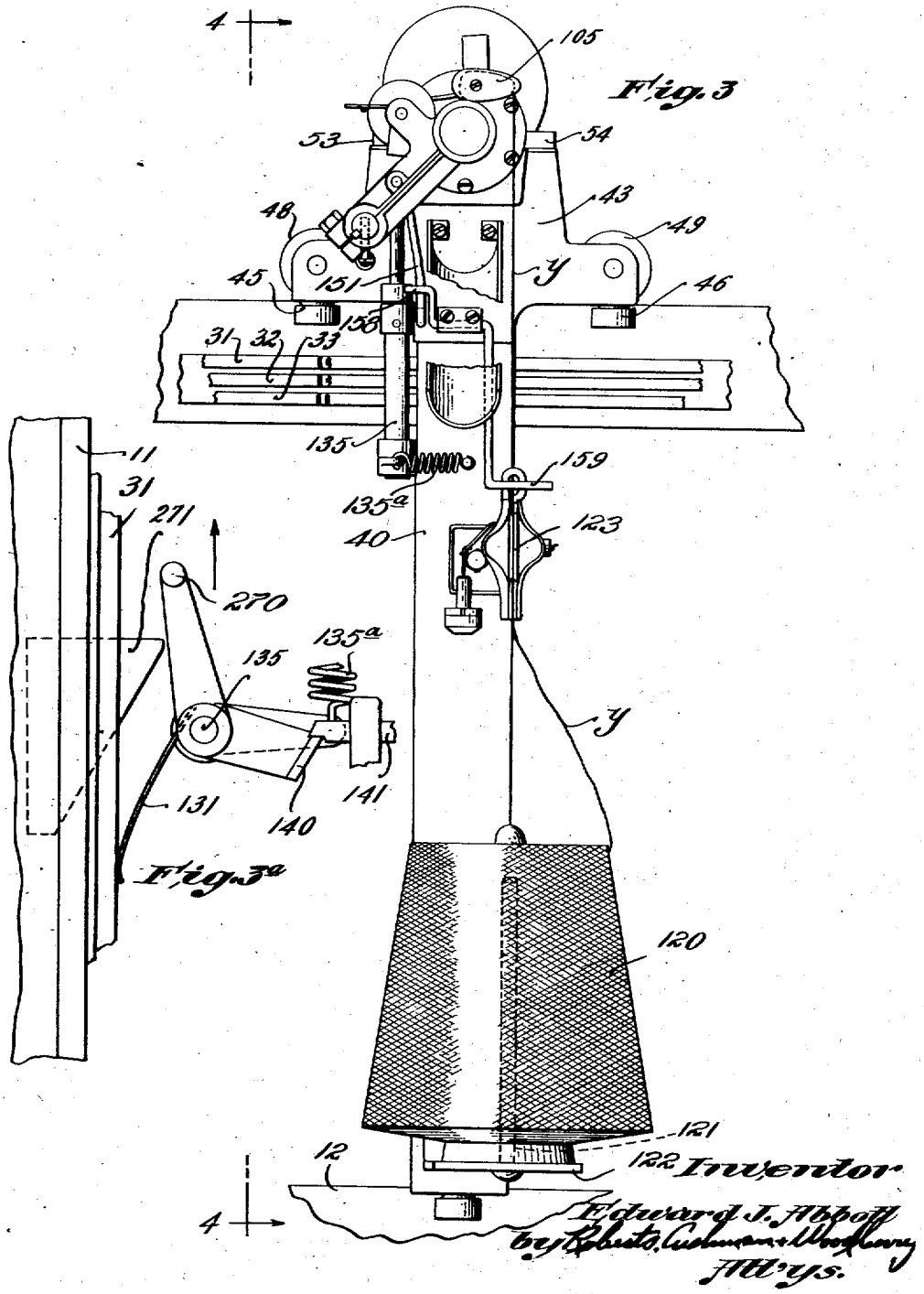


Fig. 3

Fig. 3a

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WINDING MACHINE

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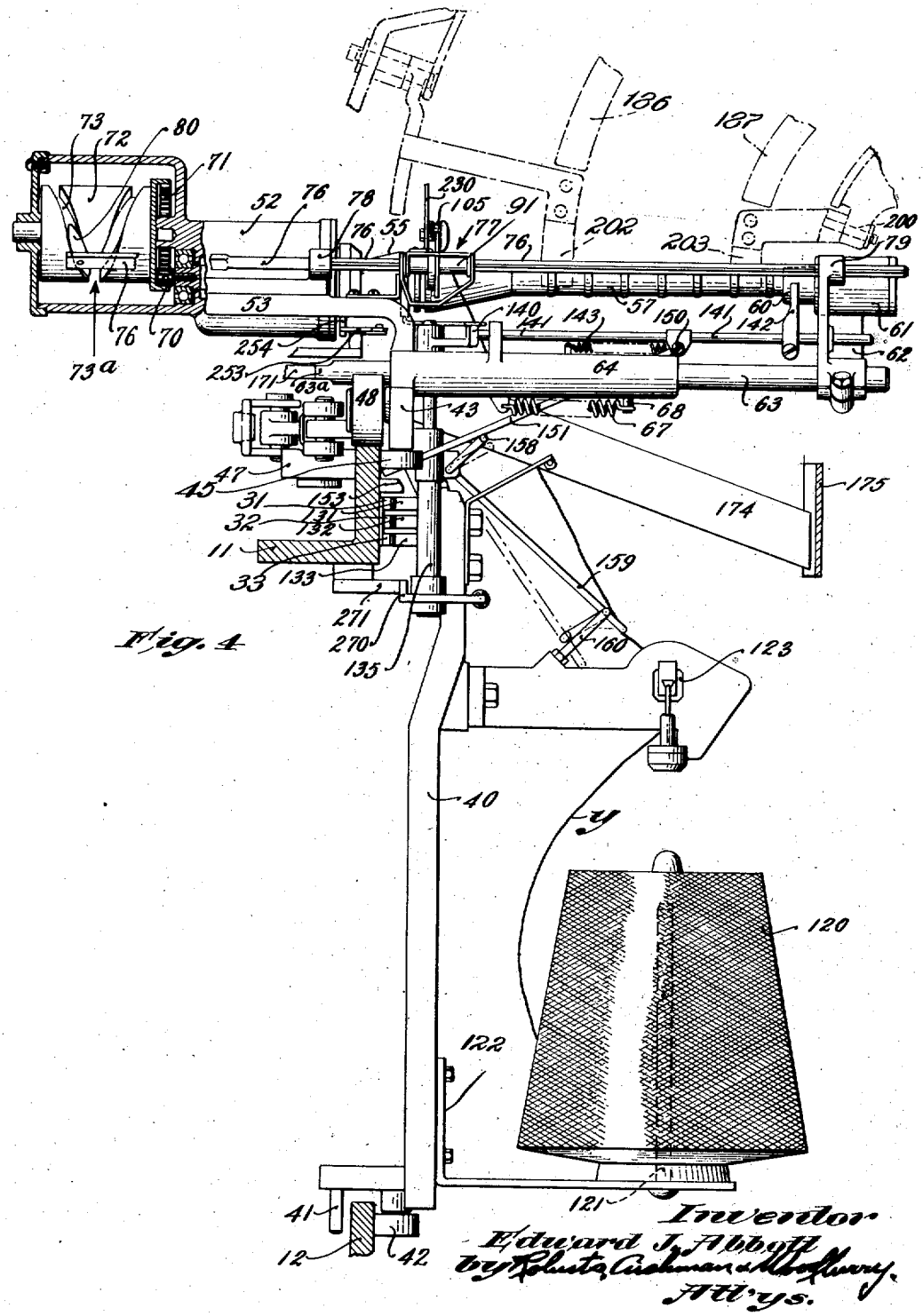


Fig. 4

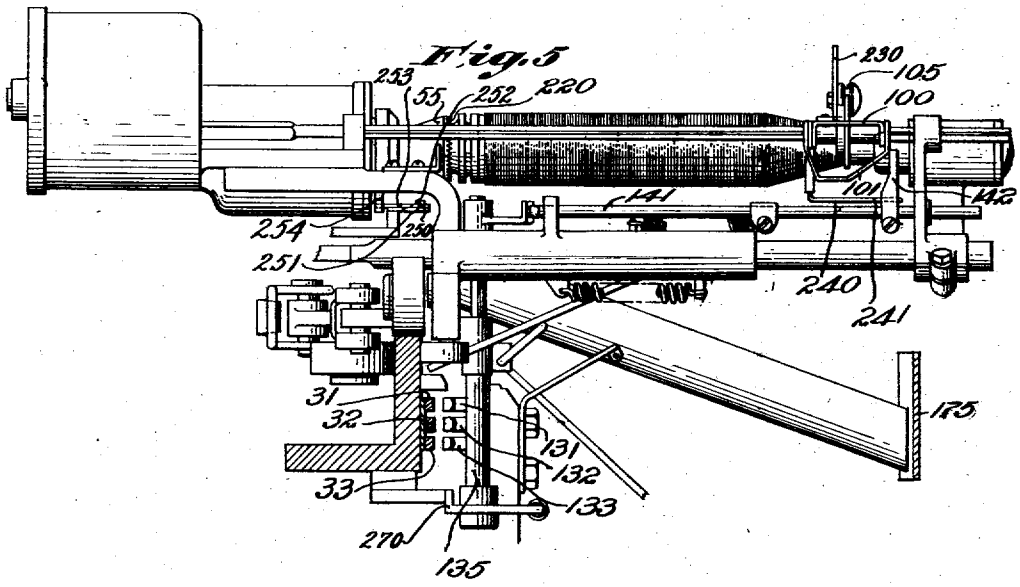
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June 6, 1939. E. J. ABBOTT 2,160,810

WINDING MACHINE

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Fig. 5



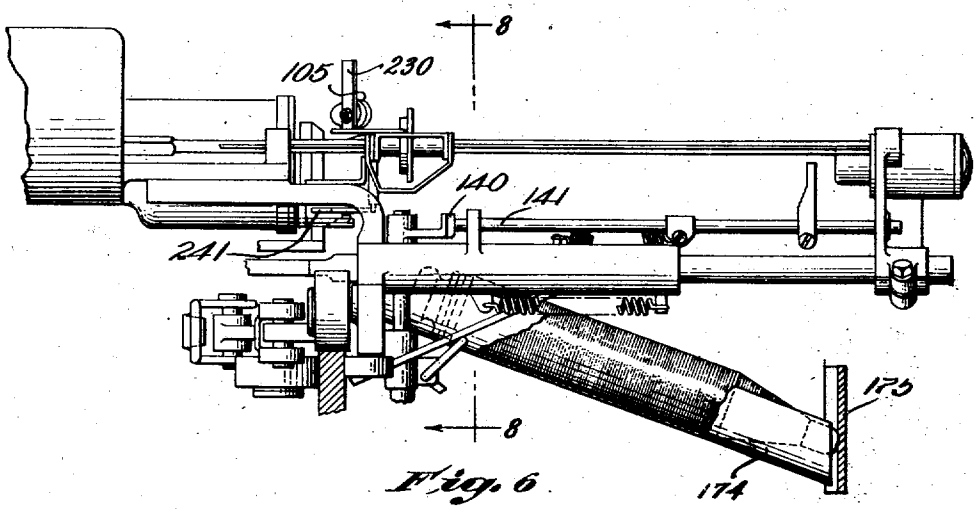


Fig. 6

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WINDING MACHINE

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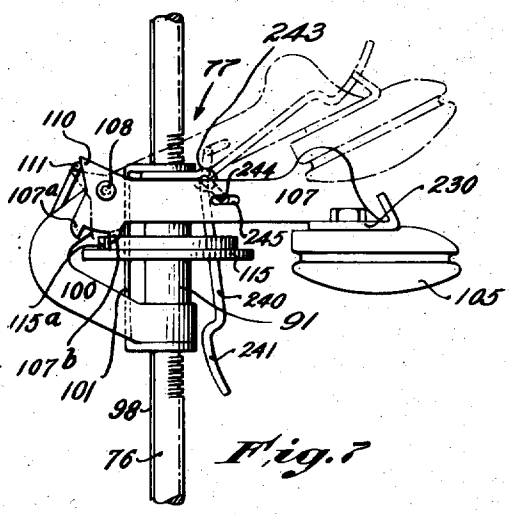


Fig. 7

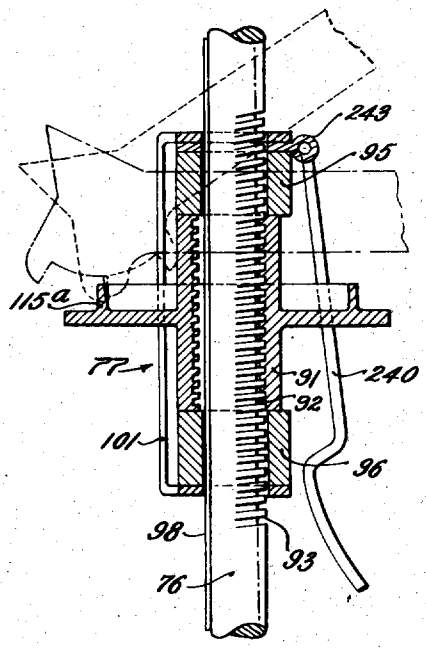


Fig. 7a

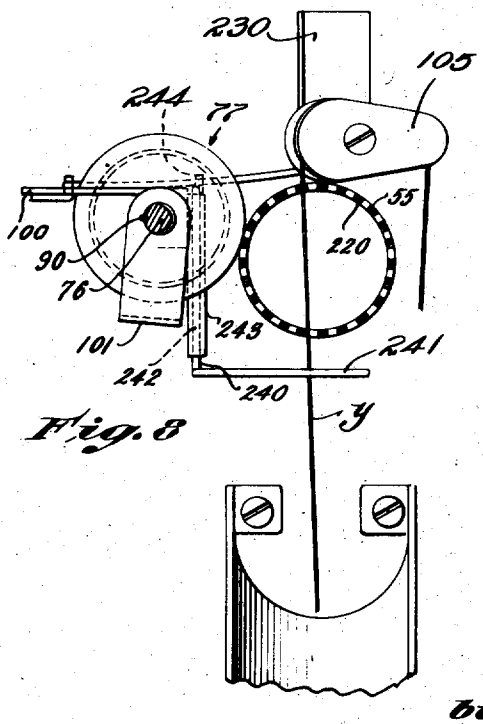


Fig. 8

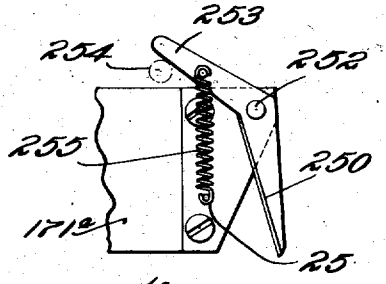


Fig. 12

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WINDING MACHINE

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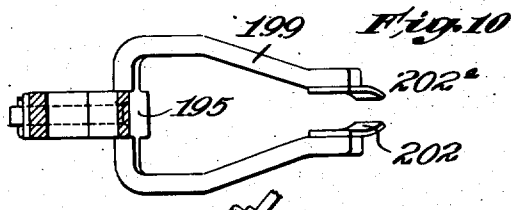
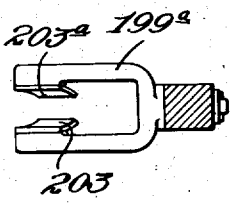


Fig. 10



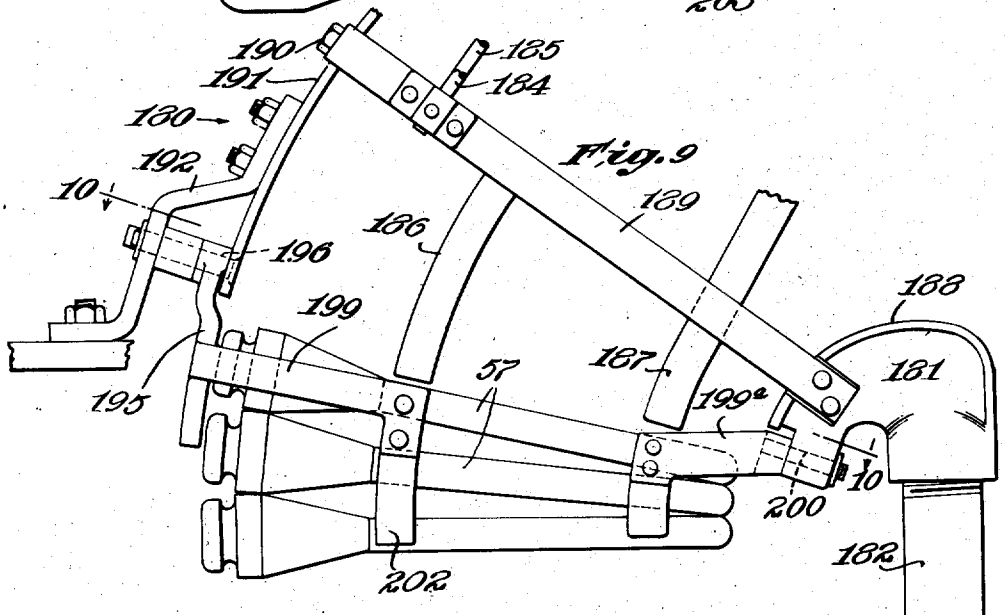


Fig. 9

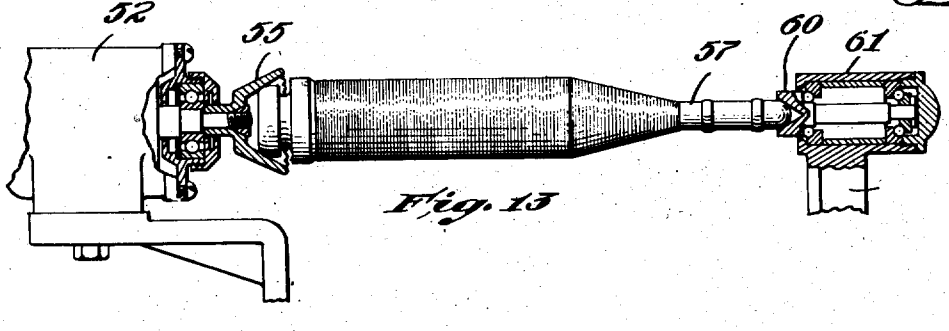


Fig. 13

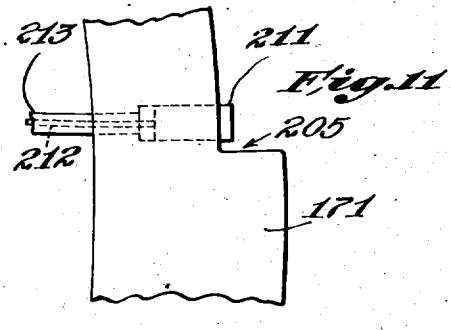


Fig. 11

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Patented June 6, 1939

2,160,810

UNITED STATES PATENT OFFICE

2,160,810

WINDING MACHINE

Edward J. Abbott, Wilton, N. H., assignor to Abbott Machine Company, Wilton, N. H., a corporation of New Hampshire

Application November 24, 1931, Serial No. 576,978

41 Claims. (Cl. 242—35.5)

This invention relates to winding machines and has for some of its principal objects to provide a winding machine wherein a new core is automatically substituted for a completed package and wherein winding is automatically begun upon the replenished core without the necessity for any prewinding of such core or any uniting or tying of one strand to another; a further object is to provide an automatic machine in which these operations are performed on a series of traveling winding carrier units which progress in a procession around the machine.

Other principal objects of the invention are to provide a winding machine in which the winding packages travel in a predetermined path in relation to the machine, usually in a procession, and wherein the winding strands are wound on the several packages in overlapped layers which advance axially of their respective packages; and to provide automatic mechanism competent to control the movement of traversing yarn guides in relation to the several traveling packages so as to perform the above described process of winding.

Other objects of invention and features of novelty and utility will be apparent from this specification and its accompanying drawings in which my invention is explained by the description of one illustrative embodiment thereof.

Fig. 1 is a side elevation of a machine embodying the present invention, with the winding units of the machine omitted;

Fig. 2 is a plan of the machine of Fig. 1, showing one of the several winding units of the machine, the other winding units and a part of the core magazine being omitted;

Fig. 3 is a fragmentary elevation of one of the winding units of the machine;

Fig. 3ª is a fragmentary diagram in plan, showing the mechanism which acts to reset the brushes into engagement with the conductor rails;

Fig. 4 is a vertical section taken at right angles to Fig. 3, for instance on the line 4—4 of Fig. 3, showing in outline the relative positions of core-supply and traveling core-holding instrumentalities; the winding unit having just received a fresh core and being in the act of beginning winding thereon;

Fig. 5 is a vertical section like Fig. 4, showing the winding unit at the time of completion of a wound package on the core and after stoppage of winding, and just prior to the doffing of the wound package;

Fig. 6 is a similar view of the unit at the time of doffing of the wound package and placing of the supply strand in position to be engaged by a new core;

Fig. 7 is a fragmentary plan view of the traverse guide and the strand-positioning arm of a winding unit, showing in dotted lines these same parts in the relative positions assumed in positioning the strand for engagement by a fresh winding core;

Fig. 7ª is a horizontal section similar to but on a larger scale than Fig. 7, showing particularly the disengageable driving connection between the traverse guide and its driving rack;

Fig. 8 is a vertical section on the line 8—8 of Fig. 6, showing the traverse guide and strand-positioning arm after completion of a wound package and just before engagement of the strand by a fresh winding core;

Fig. 9 is a fragmentary elevation of the core magazine of the machine;

Fig. 10 is a section on the line 10—10 of Fig. 9;

Fig. 11 is a fragmentary plan view illustrating a part of the mechanism for stopping travel of the carrier units upon the detection of failure to supply a fresh core to a carrier;

Fig. 12 is a fragmentary plan view showing mechanism for severing supply strands from completed wound packages; and

Fig. 13 is a vertical section through the hollow centers of a winding unit, showing a core retained for winding.

The machine illustrated herewith exemplifies an improved mode of textile strand manipulation characterized by causing a winding strand package to trave' in a predetermined path in relation to the machine while winding a strand on the package in overlapped layers which advance axially of the package.

Typically this operation is performed simultaneously in respect to a large number of winding packages; a procession is thus formed of packages in all stages of development from a freshly fed bare core to a completely wound package ready for doffing. As illustrative of the winding performed on any one of the traveling packages, the winding strand may be caused to traverse a portion of the package to form a layer of short length, whereupon the range of traverse is automatically shifted axially of the package and another like layer wound. The illustrated machine of the present invention provides for the automatic supply of empty cores to traveling core-holders at a predetermined part of their path about the machine, provides for automatically associating supply strands with the serially replenished core holders, provides for automatically

starting winding of the strands upon their respective traveling cores to form packages thereon, and preferably provides for causing (during the travel of the winding packages) relative traverse between winding packages and winding strands competent to effect the above described mode of winding on each package. Upon completion of winding of any of the traveling packages the machine is capable of automatically stopping further winding thereon; capable of depositing the wound package at a designated delivery point; and capable of severing the wound package from its supply.

Preferred form of guideway and winding unit conveying devices

Referring to Figs. 1 and 2, a suitable guideway for the various winding units of the machine may comprise upper and lower rails **11** and **12**, each disposed in opposite straight and semi-circular runs, (Fig. 2) and supported by transverse frame sections **13**, and longitudinal end frame sections **14** and **15**, (Fig. 1).

Suitable means for conveying the several winding units of the machine along the rails **11** and **12** may comprise an endless chain **14'** (Figs. 1 and 2) running on opposite sprocket wheels **15** and **16**. Either of these sprocket wheels may be driven by any suitable connections, such for example as those shown in my Patent No. 1,609,639; as illustrated herein the sprocket wheel **16** is driven by an electric motor **20**, which is connected thereto by means of a belt **21**, pulley **22**, (Fig. 2) a set of reduction gearing indicated at **23**, and a pinion **24** meshing with internal gear teeth **25** on the sprocket wheel, as more fully explained in my copending application, Serial No. 542,011, filed June 4, 1931. A suitable switch **27**, having buttons **28** and **29** for starting and stopping respectively, is adapted to control the motor **20** and hence the progress of the winding units about the machine.

Preferably the several traveling winding units derive their power for winding from individual motors traveling therewith about the machine. With this in view, the upper rail **11** of the machine frame preferably carries electric conductor rails **31**, **32**, **33**, herein shown as three in number, suitably insulated from the frame and adapted to supply current to the traveling units, as described in my copending application, Serial No. 508,868, filed January 15, 1931, Patent No. 1,964,715.

Preferred general organization of a traveling winding unit

An individual traveling winding unit preferably includes, like the machines of my above patent and copending applications, a main frame member **40** (Fig. 4) adapted to bridge the distance between upper and lower guide rails **11** and **12**, confined to the lower rail by a pin **41** and roll **42**, and carrying at its top a head member **43**. Upon this head member **43** are preferably provided horizontally disposed spaced outer rolls **45** and **46** (Fig. 3) and an inner roll **47** (Fig. 4) adapted respectively to run on opposite faces of the upper rail **11**, and the whole unit is movably supported on this rail by means of spaced vertically disposed rolls **48** and **49** (Fig. 3) running on the top edge of the rail.

An electric motor **52** is preferably mounted upon two spaced horizontal inward extensions **53** and **54** (Fig. 2) of the head member **43**. Current for operating the motor **52** is preferably conducted thereto through brushes **131**, **132** and **133** (Figs. 4 and 5) carried by a rotatable shaft **135** on the carrier and adapted to press against the stationary conductor rails **31**, **32** and **33**. From the brushes **131**, **132** and **133**, suitable wires, not shown, extend to the motor. The motor **52** is arranged to rotate the winding package, a preferred driving connection to such package consisting of a hollow center **55** (Figs. 5, 8 and 13) fast on the motor shaft and shaped to receive one end of a winding package core. Such core may be for example the typical filling or weft bobbin indicated at **57** in Fig. 4.

The other or outer end of the core is preferably held in a revolving hollow center **60** mounted in ball bearings retained in a housing **61**. Means is preferably provided for moving the outer hollow center **60** toward and from the inner center **55** to grasp and to release the cores successively handled by the unit. As shown in Fig. 4 the housing **61** for the outer center **60** is fastened by a bracket **62** to a rod **63** which is mounted for reciprocation in a sleeve **64** fast to the head portion of the carrier unit. A spring **67** acting on a pin **68** which projects down from the rod **63** through a slot (not shown) in sleeve **64** urges the rod **63** and outer housing to the left in Fig. 4, thus gripping the core **57** for winding. The inner end **63a** of rod **63** preferably projects from the left of sleeve **64** in position to be engaged by actuating means (explained hereinafter) for controlling the position of rod **63** and center **60** so as to grasp and release the cores at predetermined positions of the carrier unit in its path.

The traverse mechanism, such as will now be described, is preferably arranged to perform its strand distributing operation without imparting any substantial twist to the strand. Preferably this traverse mechanism derives its power from a pinion **70** on the motor shaft, said pinion meshing with an internal gear **71** on the end face of a traverse cam **72**. The casing of motor **52** may be extended, as shown, to enclose the cam **72**. To provide a rapid traverse with a cam of small size, this cam is preferably of the crossed groove type, having a groove **73** in the form of reversed spirals crossing at **73a**. A rod **76**, carrying a traverse assembly **77** and mounted to slide in guides **78** and **79**, has mounted at its left end a cam follower **80** adapted to run in the groove **73** of the traverse cam; operation of these elements imparts to the traverse assembly **77** a reciprocating movement along the axis of the rotating core **57** within a range equal to the throw of the cam **72**. In the form of the device illustrated herein, the traverse assembly moves from one end of its relatively rapid reciprocating stroke to the other during the time required for twenty revolutions of the core **57**.

Means is preferably provided for automatically shifting the range of movement of the traverse assembly to make the wound package consist of overlapped axially advancing layers, such means preferably acting in response to growth of the winding package. A preferred form of such means will now be described by reference to the construction of the illustrated traverse assembly **77**. Referring to Figs. 7, 7a and 8, the traverse assembly **77** shown therein includes a central rotatable tubular part **91** having internal threads **92** adapted to engage corresponding half-threads **93** on the reciprocating rod **76**. At opposite ends of tubular part **91** are two non-rotatable elements **95**, **96** each cut away to engage a key **98** on the reciprocating rod **76**. These two non-rotatable elements are fastened together by upper and lower

yokes 100 and 101. The central rotatable tubular part 91 of the traverse assembly is provided with an enlargement in the form of a wheel 115 adapted to engage the surface of the last-wound layer of strand on the winding package when the traverse assembly attains the inner or rear point in its stroke. Contact of the wheel 115 and the rotating package in this way rotates the wheel 115, causing its associated inner rotatable member 91 to turn a small amount on the threaded rod 76, thus advancing the whole traverse assembly slightly. This manner of advance of the traverse assembly is repeated each time sufficient strand is wound on a given layer. A wound bobbin is thus built with a very uniform diameter; the "build" or advance of range of the traverse guide is automatically regulated by the growth of the package regardless of the count of yarn and speed of winding.

The traverse yarn guide is herein shown as comprising a porcelain knob 105, which is disposed over the winding core 57 in position to receive a supply strand Y from beneath the core, conduct the strand above the core and to deliver it downwardly to the forming winding on the core.

The porcelain knob 105 is preferably held on the outer end of an arm 107 which is pivotally mounted at 108 on the upper yoke 100. An enlargement 110 on arm 107 is normally acted on by a spring 111 to hold the arm and porcelain 105 in the full line positions of Fig. 7. The porcelain guide 105 is capable of vertical movement such that it can ride upon the surface of the winding package and is held there by its own weight and by the pull of the strand passing over it.

The supply of strand for winding on a core is preferably carried on the individual traveling carrier unit in the form of a wound package 120 retained on a suitable peg, spindle or other holder 121 carried on a bracket 122 extending outward from the lower part of the carrier frame member 40 as shown in Fig. 4. From the supply, the strand passes upward through a suitable tension device 123 to the traverse guide 105.

Automatic stoppages of winding

(a) *Upon completion of full package.*—The rotatable rod 135 which carries the brushes 131, 132 and 133 is urged by a spring 135a (Fig. 3a) to rotate in a counterclockwise direction (viewed in plan), so as to disengage the brushes from their respective rails 31, 32 and 33, but is normally held against such movement by a reciprocable rod 141 which engages a latch member 140 at the top of the rod 135. (Figs. 3a and 4). Reciprocable rod 141 carries, near its right or outer end, an adjustable abutment 142 positioned thereon so that may be engaged by the traverse assembly 77 when the latter nearly reaches the right end of its travel which occurs when the winding operation is substantially completed. Movement of the abutment 142 to the right by the traverse assembly 77 moves the rod 141 against the action of a spring 143, (Fig. 4), disengaging the rod 141 from latch member 140 and permitting rotatable rod 135 to remove the brushes 131, 132 and 133 from the rails 31, 32 and 33, thus stopping the motor 52. Thus during the travel of the carrier unit about the machine the winding operation is automatically stopped when the winding package attains a predetermined size.

(b) *Upon breakage or exhaustion of the supply strand.*—The reciprocable rod 141, which acts to release rotatable rod 135 for stoppage of the motor, has pivoted thereon at 150 (Fig. 4) a rod 151 of a length sufficient to engage any one of a series of cams 153 (Figs. 1, 2 and 4) disposed at intervals along the upper guide rail 11. As indicated in Fig. 2, each of these spaced cams 153 increases in width in the direction of travel of the carrier units; the effect of a cam 153 is to force outwardly the rod 151 of any of the carrier units engaged thereby, thus moving the reciprocable rod 141 to the right, releasing the rotatable rod 135 and stopping the motor 52 in the same manner as in stopping winding of a full package. The rod 151 is normally held clear of the cams 153 by a crank 158 (Figs. 3 and 4) forming part of a pivoted detector wire 159 which rests on the strand running from the tension device 123 to the traverse guide 105. Upon breakage or exhaustion of the strand, detector wire 159 drops to the dotted position in Fig. 4, allowing rod 151 to drop into position for its end to be engaged by the next adjacent cam 153, whereupon stoppage of the motor 52 occurs as just explained.

(c) *At the end of a predetermined winding interval.*—As is more fully explained below, winding on each of the traveling carrier units commences as soon as the individual unit has passed around the right-hand end of the machine. Means is provided for automatically stopping winding on any of the the traveling units severally at the end of a predetermined interval, which in the case of the present machine is the time required for the individual unit to pass around the machine and return again to the right-hand or replenishing end. By the provision of such means winding ceases at this point, even though the other stop devices described above have not as yet operated.

In the illustrated machine winding may be stopped at the end of a predetermined interval by an arrangement of the current conductors 31, 32 and 33, such as shown in Fig. 1, wherein these conductors end near the right-hand end of the machine; upon reaching the end of these conducting rails, the motor 52 of the traveling carrier unit is deenergized and remains inactive until the carrier unit again reaches the start of these same rails at a point indicated at 31a (Fig. 2) just before a starting cam 271 described below. If desired, the rails 31, 32, 33, may merely be provided with deenergized sections extending around the right-hand end of the machine to this point 31a in a manner similar to that disclosed in my copending application Serial No. 508,868, filed January 15, 1931.

Automatic holding and releasing of winding package core and collection of full packages

It has been described above how, during its travel along the guide rails, each unit holds a winding package core 57 between a live center 55 and an outer dead center 60, which later is carried by bracket 62 on a reciprocable rod 63. Means is provided for automatically causing the carrier unit to release its core 57 at a predetermined point in its travel. As indicated in Figs. 1, 2 and 4, a fixed vertical post 170, coaxial with the sprocket 16, may carry a cam 171 of such shape and position as to engage the inwardly extending end 63a of the reciprocable rod 63 of each carrier. Upon contacting with this cam 171, the reciprocable rod 63 of the carrier is forced outwardly, (to the right in Fig. 4) carrying the bracket 62 and outer center 60 with it, thus releasing the winding package core 57. This operation will take place whenever the carrier unit passes slightly beyond the position of the single unit shown in Fig. 2.

It will be observed that upon release of the core

57 (usually carrying a full wound package) the strand extending from the supply 120 is still unbroken. Means is preferably provided for detaining the released core and package on the traveling carrier during the time necessary to associate the strand with new core for resumption of winding. In the illustrated machine, such means is shown as comprising a package receiver in the form of a trough 174 disposed directly beneath the normal winding position of a core and preferably sloping downwardly towards its outer end in the form of a chute as shown in Fig. 4. Cooperating with these troughs or chutes 174 of the several carriers there is preferably provided an arcuate plate 175 (Figs. 1, 2 and 4) disposed to close the outer lower ends of the troughs until the carriers reach a point somewhat beyond the extreme semi-circular end of the machine. Upon reaching the end of this guard plate 173, a full package or partially wound core in the trough 174 of the carrier is free to slide out the end of the trough and into a receptacle 176 suitably positioned. Thus the machine provides for delivering all of the released wound packages at a predetermined part of the travel of the several units.

Automatic supply of winding package core

The means provided in the present machine for holding and supplying fresh winding cores to the carrier units is shown as comprising a magazine 180 (Fig. 1) which may conveniently be mounted in part at least upon a bracket 181 carried by a vertical support 182, the magazine being positioned approximately in prolongation of the longitudinal axis of the machine. The exact construction of the magazine will of course vary with the particular kind of winding core to be employed by the machine; for the type of core illustrated the magazine preferably utilizes inclined rods 184, 185, spaced at a suitable distance to hold the cores by their larger head ends with their smaller ends depending as indicated in Fig. 1. The cores slide down the two rods 184 and 185 until they are engaged by spaced arcuate plates 186 and 187 on both sides and their tip ends rest upon a curved portion 188 of the bracket 181. The arcuate plates 186 and 187 are fastened together by a U-shaped strap member 189 connected at its two ends to the bracket 181. The closed end of this U-shaped strap member 189 is connected at 190 to an arcuate plate 191 which in turn is carried by a bracket 192.

The lower portion of the magazine includes an arcuate plate 195 hinged at 196 to the plate 191, and a U-shaped member 199 integral with the arcuate plate 195, (Figs. 9 and 10). Fastened on both sides of this swinging member 199 are downwardly extending spring fingers 202, 202a, each approximately in prolongation of the plates 186 just above them. The magazine includes also another U-shaped member 199a pivotally mounted at 200 on the bracket 181 so as to swing about the same inclined axis as the U-shaped member 199. Spring fingers 203, 203a fastened on member 199a extend generally in downward prolongation of the upper plates 187. The spring fingers 202, 202a and 203, 203a preferably have their lower ends slightly inturned as shown in Fig. 10, yieldingly to hold the lowermost core of a stack in the magazine, as shown in Figs. 1 and 9. The position of the magazine is such that the lowermost core of the stack thus held is in position to be cleared both by the live center 55 and the outer dead center 60, when this outer center has been forced to its outermost position by the action of the cam 171; furthermore, the lowermost core 57 of the magazine is held at the appropriate level to be engaged between these two centers 55 and 60 of a passing unit when the outer dead center is forced inwardly toward the live center 55.

For causing the lowermost core of the magazine thus to be engaged by the core-holding centers on a winding unit, the cam 171 which has previously held the dead center out, abruptly ends at 205 (Fig. 11) allowing the spring 67 to force the center 60 inward or to the left, thus grasping the core 57 firmly between the two centers. Further movement of the traveling carrier beyond this point causes the lower hinged portions 199, 199a and the spring fingers 202, 202a and 203, 203a, to swing about the pivots 196 and 200 until the core 57 is withdrawn from these spring fingers; the spring fingers retain all the remaining cores in the magazine and the hinged lower portions of the magazine swing downward into position to supply a core to the next traveling winding unit.

The pivotal mountings of the lower parts of the magazine are of advantage in allowing the relative motion of the core-holding centers 55 and 60 with relation to the magazine to withdraw a core from the magazine. In addition, such pivotal mountings allow the spring fingers 202, 202a, 203 and 203a to swing to permit the reciprocable rod 76 of the traveling unit to pass thereunder without interference. The parts 199 and 199a although mounted to swing on a common axis, preferably have freedom for independent motion about pivots 196 and 200 because of the fact that the outer tip end of a core moves on a greater radius than the inner end of the core at the curved end of the machine; the pivoted sections of the magazine which retain the two ends of the cores will therefore not move exactly together.

Automatic cessation of travel of the units upon exhaustion or stoppage of the magazine

In case the magazine 180 is empty, or in case a core 57 is not in place therein to be engaged between the centers 55 and 60 of a traveling unit, the outer center 60 and reciprocable rod 63 of the unit will travel farther than usual to the left when the end 63a of rod 63 is released by the end 205 of the cam 171. A push pin 211 (Fig. 11) operating a piano wire 212 in a flexible conduit 213 is disposed immediately beyond the end 205 of the cam 171. This push pin 211 is normally cleared by the end 63a of the rod 63 when movement of the rod 63 has been checked by the engagement of a core 57. Movement of the rod 63 to a greater distance to the left when the unit fails to engage a core forces this push pin 211 and the piano wire 212 inward. The opposite end of the piano wire carries a pin 214 (Fig. 1) disposed in position to strike and operate the stop button 29 of the push button switch 27. This mechanism thus detects the failure of the magazine to supply a fresh core to a traveling unit, and upon the detection of such failure deenergizes the motor 20 and thereby stops the traveling motion of the various carrier units. In addition to preventing idle movement of the machine in the absence of an available supply of fresh cores this mechanism also serves to prevent any carrier unit from traveling far enough to discharge its wound package until a fresh core has been placed in winding position in the unit. Exhaustion of the supply of winding cores therefore does not necessitate a rethreading of each of the carriers such as would be necessary were

the carriers to continue discharging their wound packages after an exhaustion or stoppage of supply of fresh cores.

Automatic association of the supply strand with the fresh winding core

The machine in its preferred forms is capable of automatically associating the supply strand which travels with the given carrier with the fresh core without the necessity for any prewinding on the newly supplied core or for any tying of knots.

It will be observed that the location of the porcelain guide eye **105** is such that when the bobbin is released from its centers and dropped into the trough **174**, this porcelain eye **105** retains the run of yarn extending from the tension device **123** to the wound bobbin; it is preferably this portion of yarn extending from the porcelain guide eye **105** to the completed released bobbin which is associated with the newly supplied core; thus the traverse guide eye **105**, tension device, and any other instrumentalities acting on the strand need not be rethreaded. Means is therefore provided for carrying the portion of yarn which extends from the guide eye **105** to the released bobbin over to the left in Fig. 6 to a position wherein this strand can be associated with a new core inserted between the centers **55** and **60**.

Preferably the hollow center **55** which engages the head of the core is provided with means for imparting a winding rotation to such a strand presented thereto so that this portion of the strand, placed in engagement with the hollow center **55**, will be caused to wind up upon a core newly placed in the center **55**. As shown in Figs. 5 and 8, the center **55** is preferably provided around its rim with toothed serrations **220** forming notches across which the yarn Y may be laid as shown in Fig. 8. A strand thus laid across the serrations in the hollow center **55** will be so positioned as automatically to be gripped between the center **55** and a new core **57** when the core **57** is forced endwise against the center **55** by the outer center **60**. In this way the strand leading from the supply **120** can be automatically associated with the new core.

It will be understood that traverse assembly **77** of the unit progresses relatively slowly toward the right during winding and at the completion of winding attains approximately the position shown in Fig. 5. The illustrated machine includes mechanism for automatically restoring this traverse assembly by a rapid return movement to a predetermined position at the opposite end of its travel, and the mechanism employed for this purpose preferably also serves automatically to carry the strand extending from the porcelain guide **105** over into engagement with the serrations **220** of the hollow center **55**. As previously described the traverse assembly includes an arm **107** pivoted at **108** and normally held in the full line position of Fig. 7 by a spring **111**. This arm **107**, which serves to hold the porcelain guide **105**, preferably includes an upturned portion **230** (Figs. 5, 6 and 7) adapted to cooperate with a stop cam **231**, (Figs. 1 and 2) for moving the traverse assembly to the left. This cam **231** which may conveniently be carried by arms **233** and **234** extending from the vertical post **170** is so positioned as to engage the upstanding extension **230** on the arm **107** soon after the full package has been released by the center **60**. Further movement of the carrier around the end of the machine turns the arm **107** to its dotted position in Fig. 7 and moves the entire traverse assembly **77** to its extreme left-hand position on the rod **76** by this engagement of the extension **230** with the cam **231**. In order to release the traverse assembly from the threads **93** on the rod **76** during this movement to the left the pivoted arm **107** preferably carries a shoulder **107a** (Fig. 7) adapted to strike a rim **115a** of the wheel **115**. When the extension **230** is struck by the cam **231**, and the arm **107** thus swung to the dotted line position of Fig. 7, the portion **107a** of this arm striking the rim **115a** forces the wheel **115** to the right in Fig. 7 transversely of the rod **76**, and disengages the threads **92** on the tubular part **91** from the corresponding threads **93** on the rod **76**. The traverse assembly is thus free to slide along the rod **76** under the influence of the cam **231**. Upon arrival at the left-hand end of the rod **76** the arm **107** and its attachments still occupy the relative position shown in dotted lines in Fig. 7; in this way, the porcelain eye **105** and the upturned extension **230** are moved past the serrated hollow center **55** and clear the magazine **180** and its retained bobbins.

This movement of the traverse assembly to the left is preferably utilized to associate the strand Y extending down from the porcelain eye **105** with the serrated hollow center **55**. For this purpose the traverse assembly preferably carries a wire **240** having a horizontally disposed arcuate portion **241** (Fig. 8) below the level of the hollow center **55**, and a vertical portion **242** pivotally mounted to turn about a vertical axis in a loop **243** carried by the traverse assembly. At its upper end the wire **240** is provided with a crank arm **244** which is slidably engaged in a slot **245** in the arm **107**. The relations of these parts are such that when the porcelain eye **105** occupies its full line position in Fig. 7, the wire **240** is clear of the path through which a full package drops in being released by the centers **55** and **60**. When the arm **107** is swung to its dotted line position in Fig. 7 by the action of the cam **231**, the wire **240** swings through an arc of about 120°, as shown by the dotted position of this arm. (Fig. 7). During this movement the hook-shaped outer portion **241** of the wire engages the strand which now extends downward from the porcelain eye **105** to the released package. Movement of the traverse assembly to the left along the rod **76** (Fig. 4) causes the arm **240** to carry the yarn Y to the left; the porcelain eye **105** and the wire **240** engaging the yarn above and below the serrated hollow center **55**, respectively, place the yarn across this center as shown in Fig. 8.

After this sequence of operations the traveling unit reaches the magazine **180**, at which point a fresh bobbin **57** is grasped between the centers **55** and **60** as described above, the head end of the bobbin securing and clamping the strand against the live center **55**.

Automatic severing of the supply strand from the wound package

Automatic means for cutting off the yarn of the wound package from the portion of the supply strand now clamped is herein shown as comprising upper and lower scissor blades **250** and **251** (Figs. 2, 5 and 12) suitably mounted, for example upon an extension **171a** of the cam **171**, in position to engage between them a portion of the strand extending down from the wire **240** to the released full package when the wire **240** is at its extreme inner or left-hand position. Upper blade

250 is preferably pivoted to the lower blade 251 at 252 and carries integral therewith an inwardly extending crank arm 253. The head portion of the carrier is provided with a depending pin 254 disposed in position to engage this arm 253 in passing and to turn the upper blade 252 so as to sever the yarn. A suitable spring 255 then returns the scissors to their open position ready to operate upon the yarn of the next traveling carrier unit.

Resetting of traverse mechanism in position to wind

After clamping the strand between the new core and the hollow center 55 and severing the completed package from this strand, the traverse guide 105 is automatically carried over to the yarn receiving portion of the core in position for winding to commence. For this purpose there is preferably provided a cam 260 (Fig. 2), conveniently mounted integral with one of the supports 234 of the cam 231,—this cam 260 projecting into the path traversed by the upper projection 230 of the traverse assembly. Upon contacting with this cam 260, the projection 230, the arm 107 and porcelain guide eye 105 are returned to positions such as are shown in full lines in Fig. 7. The cam 260 is so positioned with respect to the position of a new core 57 as to leave the porcelain eye 105 at approximately the head end of the core in position to start a layer of winding at this end. Meanwhile the end of the yarn Y which extends over the porcelain eye 105 remains anchored between the end of the core 57 and the hollow center 55 so that when the hollow center 55 begins to turn the yarn is drawn on to the rotating core.

When the arm 107 and eye 105 are returned to their full line positions in Fig. 7 in position to start winding, a projection 107b on arm 107 contacts with an inner surface of the rim 115a on the wheel 115 and urges the associated threaded barrel 91 into engagement with the half-threads 93 on the rod 76. The traverse assembly is thus fastened to the rod 76 so as to be moved therewith as the rod 76 reciprocates under the influence of the cam 72 in winding.

Resetting of stop motion mechanism and resumption of winding

When the carrier encounters the live conductor rails 31, 32 and 33 on the opposite side of the machine from that shown in Fig. 1, a resetting mechanism acts to restore brushes 131, 132, 133 into position such as shown in Fig. 4, in contact with these rails. It will be understood that in the case of most of the carrier units, the rod 135 will previously have been rotated (either as a consequence of breakage or exhaustion of the yarn or the completion of a full package) so as to remove the brushes from contact with their rails. To restore the brushes into contact with the rails the crank arm 270 is provided on the lower end of rotatable rod 135 in position to engage a cam 271 located at the beginning of the conductor rails 31, 32 and 33 on the far side of the machine. When the crank arm 270 encounters this cam it is rotated thereby in a direction opposite to that induced by the spring 135a. This rotates the vertical rod 135 and restores the brushes into position to contact with the conductor rails. The latch member 140 which has occupied a position behind the rod 141, (when these parts are viewed as in Figs. 4, 5 and 6) now forces this rod 141 slightly to the right by a camming action and passes its end allowing this rod to snap back behind it, the latch member 140 now being in front of the tip end of rod 141 as shown in Figs. 4 and 3a. The interengagement of rod 141 with latch member 140 thus locks the brushes 131, 132 and 133 in their operative positions, in contact with rails 31, 32 and 33.

By the time the crank arm 270 leaves the cam surface 271 the motor 52 will have started, the strand will be winding upon the fresh core 57, and the portion of the strand engaged by the detector arm 159 will be under tension so that the brushes 131, 132 and 133 will not again be disengaged until some condition of breakage or exhaustion of the strand occurs or a full package is completed. The last stop-motion cam 153 on the side of the machine shown in Fig. 1 is preferably extended around the curved end of the machine to a point 153b just beyond the starting cam 271 in the form of a ledge 153a on top of which the stop-motion rod 141 can ride. This rod 141 is thus supported during the time in which there is no winding tension on the supply strand.

Since the various operations incident to the automatic control of winding and to the automatic replenishment and initiation of winding have been described for one of the several traveling carrier units, no further description of the operation of the machine as a whole is deemed necessary; it will be understood that the various controlling and replenishing operations are performed with relation to each of the several carrier units, the carrier units being subjected in regular order to the several replenishing operations.

It should be understood that the present disclosure is for the purpose of illustration only and that this invention includes all modifications and equivalents which fall within the scope of the appended claims.

I claim:

1. A textile machine having therein a carrier adapted to carry a winding strand package core and an unwinding supply strand mass, means for moving the carrier in a predetermined path in relation to the machine, means for winding a strand from the supply strand mass onto the winding strand package core during such motion, means for distributing the strand on the core in progressively axially advancing conical layers of less length than the finished package, in combination with means operative after winding of a package on the core for automatically associating the strand of a supply mass remaining on the carrier with a fresh core for initiating the winding of a new strand package on the carrier, means for supplying a fresh core to the carrier, means for doffing a wound core from the carrier, and means for positioning the strand-distributing means for the beginning of a new winding on a fresh core.

2. A textile machine having therein a carrier adapted to carry a winding strand package core and an unwinding supply strand mass, means for moving the carrier in a predetermined path in relation to the machine, means for winding a strand from the supply strand mass onto the winding strand package core during such motion, means for doffing the said package from the carrier, means for automatically providing a fresh winding package core on the carrier, and means for associating the strand from the supply mass remaining on the carrier with such core for winding thereon.

3. In a winding machine, a rotatable center

recessed to receive an end of a winding package core, the part of said center surrounding the recess having serrations to receive a strand laid across said recess.

4. In a winding machine, a rotatable center recessed to receive an end of a winding package core, and having means for detaining a strand in position to be secured between the center and a core held thereagainst.

5. In a winding machine, rotatable means for catching and instituting winding of a strand, and a hollow center rotatable with said means and recessed to receive and position the end of a core in contact with said strand.

6. A winding machine having therein means for holding a winding package core for rotation, said means including a hollow rotatable center recessed to receive and embrace one end of said core, said hollow center being shaped to engage and detain a strand presented thereto to institute winding of the strand, and automatic means for presenting a strand to said center.

7. A winding machine having therein means for holding a winding package core for rotation, said means including a rotatable hollow center recessed to receive one end of said core, means for doffing a core and a package wound thereon from said holder, and automatic means for laying the strand from the doffed package across said hollow center.

8. A winding machine having therein means for holding a winding package core for rotation, said means including a rotatable center recessed to receive one end of said core, means for doffing a core, and a package wound thereon from said holding means, and a pair of strand-engaging parts movable toward the hollow center on opposite sides thereof and engageable with the strand leading to the doffed package to lay said strand across the hollow center.

9. A winding machine having therein a holder for rotatably supporting a winding package core, core-supplying means adapted to cooperate with said holder to furnish new cores for the winding of successive packages, said holder and said core-supplying means being arranged with respect to each other for relative motion such that one passes the other at intervals, means for rotating a winding package core held by the holder to wind on said core, traverse means acting to distribute a winding strand on the package, said traverse means including a strand guide movable lengthwise of the package during winding in a path relatively close to the periphery of the winding package, and means for automatically imparting a supplemental movement to said strand guide in a direction such as to cause the strand guide to clear said core-supplying means when the latter supplies a new core to said holder.

10. A winding machine having therein a traversing strand guide engageable with a winding strand to distribute the strand lengthwise of a winding package core without substantially twisting the strand, means for moving said strand guide outside of its normal winding range of traverse upon completion of winding, thereby to carry the strand out of its normal path of travel, and means engageable with a strand thus carried to position the strand in association with a fresh core for winding another package.

11. A winding machine having therein a traversing strand guide engageable with a winding strand to distribute the strand lengthwise of a winding package core without substantially twisting the strand, means for moving said strand guide outside of its normal winding range of traverse upon completion of winding, thereby to carry the strand out of its normal path of travel, in combination with means for instituting winding of the strand thus carried upon a fresh core, and means for automatically moving the strand guide into traversing relation to the fresh core to distribute the strand thereon.

12. A winding machine having therein means for holding a winding package core for rotation, said holding means including a rotatable center recessed to receive one end of said core, a traversing strand guide for distributing the strand upon said core, means for doffing a core and the package wound thereon from said holding means, and means for placing on the said hollow center the strand engaged by the strand guide, said last-named means including mechanism acting to move the strand guide outside its normal range of traverse toward said hollow center.

13. A winding machine having therein means for retaining an unwinding supply mass of strand, a traversing strand guide and a winding package core holder both so disposed above the supply mass that the strand first passes upward to the traversing guide and then downward to the core held by said holder, in combination with means for discharging a wound package and its core from said holder, means for placing a new core in said holder, and means for associating the leading end of the strand remaining in the said guide with the new core.

14. A winding machine having therein means for winding a strand from a supply onto a rotating core, means for traversing the strand on the core, means for causing a progressive shifting of the range of traverse in a direction along the axis of the core during winding, in combination with means for automatically stopping such traversing upon completion of the package, means for returning the traverse means to the beginning end of a core, means for automatically associating the strand with a fresh core for winding thereon, and means for automatically resuming traversing of the strand.

15. A winding machine having therein means for rotatably holding a winding strand package core between opposed centers for winding strand on the core from a supply, thus to form a wound package, means for releasing the core and its package from the centers, the parts being so arranged that upon such release the package falls by gravity, means for detaining the filled package after release, means for detaining the strand leading from the supply to said package, and means for automatically severing the strand.

16. A winding machine having therein a holder for rotatably holding a package core for winding a package thereon from a supply, means for disengaging the core from the holder, the parts being so arranged that the core and the package wound thereon drops by gravity upon being thus disengaged, a chute disposed to intercept the package as it falls, means whereby the package is temporarily detained in the chute, and means for subsequently discharging the package from said chute.

17. A winding machine having therein a holder for rotatably holding a package core for winding a package thereon from a supply, means for disengaging the core from the holder, the parts being so arranged that the core and the package wound thereon drops by gravity upon being thus disengaged, a chute disposed to intercept the package as it falls, means whereby the package

is temporarily detained in the chute, means for severing the strand leading from the supply to the said package, and means for discharging the package from said chute after such severance.

18. A winding machine having therein a holder for rotatably holding a package core for winding a package thereon from a supply, means for disengaging the core from the holder, the parts being so arranged that the core and the package wound thereon drops by gravity upon being thus disengaged, a chute disposed to intercept the package as it falls, means whereby the package is detained in the chute, means for automatically associating the strand of the supply with a new core for winding thereon, means for severing the strand from the wound package, and means for discharging the package from said chute.

19. A machine of the class described comprising means for releasably holding a core while winding a strand thereon to form a package, means for supplying strand to be wound on the core, means for releasing the core from its holding means at the completion of winding, means for temporarily detaining the released core in proximity to but spaced from the winding position, and means for severing the strand between the package and the supply while the core is so detained.

20. An automatic winding machine having therein a group of winding units, and automatic replenishing mechanism therefor, said mechanism including a core-supplying device adapted to cooperate with the several units to supply cores thereto, and means for effecting relative movement between the said group of winding units and the core-supplying device such as to bring the said device and the several winding units into cooperative relation, in combination with means for stopping such relative movement upon failure of the core-supplying device properly to supply a core to a winding unit.

21. An automatic winding machine having therein a group of winding units, and automatic replenishing mechanism therefor, said mechanism including a core-supplying device adapted to cooperate with the several units to supply cores thereto, and means for effecting relative movement between the said group of winding units and the core supplying device such as to bring the said device and the several winding units into cooperative relation, in combination with means responsive to exhaustion or stoppage in the core-supplying device, for stopping such relative movement.

22. A winding machine having therein a plurality of holders for rotatably holding winding package cores, means for causing the holders to travel in a procession in a predetermined path in relation to the machine, means for winding upon the cores during such travel, and means, responsive to the absence of a core in a holder, for stopping the travel of said holders.

23. A winding machine having therein a plurality of winding units each including means for rotating a winding strand package, a traversing element for distributing a strand on the package, and means governed by the amount of the wound strand on the winding package for advancing the traversing element with respect to the package, the machine having in combination with said winding units means for automatically returning the several traverse elements successively to predetermined positions in relation to their respective units.

24. A winding machine having therein a plurality of winding units each including traverse devices adapted to distribute a winding strand on a package in overlapping axially advancing layers, the machine having in combination with said winding units means for automatically restoring the several traverse devices successively to starting positions for beginning winding of new packages.

25. A winding machine having therein a plurality of winding units each including a traversing yarn guide, means for moving said units in a predetermined path, means for winding yarn packages on the several units during such motion, and means for causing the several yarn guides to cross relatively slowly from one end to the opposite end of their respective winding packages while imparting a back and forth traversing motion to their respective yarn strands, in combination with means for imparting to the several yarn guides in succession a relatively rapid return movement at a predetermined point in the path of travel of said units.

26. A winding machine having means for automatically supplying a core for a winding package, a yarn guide, means for traversing the guide whereby to wind the package in a predetermined manner, the normal limits of traverse of the guide being between the ends of the core, and means for causing the guide to move beyond the end of the core at the completion of the winding operation to carry the strand adjacent to the end of a new core supplied by said core supplying means.

27. A winding machine having therein a plurality of winding units, each including opposed centers adapted to hold a core for rotation, means for simultaneously driving a plurality of said cores held in said centers to wind thereon, a magazine adapted to hold a plurality of cores to be wound, means for automatically presenting the magazine in succession to the several winding units, and means for automatically causing the centers of said units to engage and hold a core from said magazine.

28. A winding machine having therein core end-clamping mechanism acting automatically to clamp a core for winding, means for automatically supplying a core to said core end-clamping mechanism, and means operable after clamping of the core for rotating the clamped core to wind strand thereon, in combination with a traversing strand guide adapted to distribute the winding strand on the rotating core, the strand guide having a full stroke less than the length of the package, means for shifting the zone of traverse stroke of the guide axially of the core during winding, and means for automatically causing the zone of the strand guide stroke to lie at one end of the core at the start of winding thereon.

29. In a winding machine, a plurality of winding devices each including a pair of spaced centers adapted to engage opposite ends of a winding core, a magazine adapted to hold a plurality of cores, means for presenting the magazine to the several winding devices in succession in position for a core in the magazine to be engaged by the said centers of the winding device, and means for movably supporting the portion of the magazine which holds the endmost core so that said portion at least of the magazine can move out of its normal position to pass the successive winding devices.

30. A winding machine having therein a plurality of devices for rotatably holding winding package cores, means for simultaneously rotat-

ing a plurality of the cores thus held to wind thereon at the same time, a magazine adapted to hold and to discharge the cores to be wound, traveling mechanism acting automatically to bring the magazine and the several core-holding devices into cooperative relation in succession, means for causing the several core-holding devices in succession to engage a core in the magazine, the magazine being so disposed with relation to the series of core-holding devices that the said traveling mechanism causes said devices in succession to withdraw a core from the magazine.

31. A winding machine having therein a plurality of devices for rotatably holding winding package cores, means for simultaneously rotating a plurality of the cores thus held to wind thereon at the same time, means for moving the core-holding devices in a predetermined path, a magazine for cores disposed adjacent to said path, and means for causing the several core-holding devices in succession to withdraw a core from said magazine.

32. A winding machine having therein a plurality of devices for rotatably holding winding package cores, means for simultaneously winding strands on a plurality of the several cores thus held, automatic winding-core replenishment mechanism adapted to supply a fresh core to any of said holding devices, means for effecting relative traveling motion competent to associate the replenishment mechanism with the several holding devices in succession, and means for automatically starting an end of strand on each replenishment winding core.

33. A winding machine having therein a plurality of core-clamping devices, means for simultaneously rotating a plurality of the cores clamped by said devices to wind on said cores, and means for automatically causing the clamping devices to release their cores in predetermined order.

34. A winding machine having therein a plurality of core-clamping devices, means for simultaneously rotating a plurality of the cores clamped by said devices to wind on said cores, clamp releasing means, and traveling mechanism acting to bring the clamp releasing means into operative relation to the several core-clamping devices in succession.

35. A winding machine having therein a plurality of core-clamping devices, means for simultaneously rotating a plurality of the cores clamped by said devices to wind on said cores, means for moving the clamping devices and their retained cores in a predetermined path, and means for causing the clamping devices severally to release the carried core at a predetermined part of said path.

36. A winding machine having therein a plurality of winding devices, each including a pair of spaced centers adapted to engage opposite ends of a winding core, a magazine adapted to hold a plurality of cores, means for moving the winding devices in a predetermined path past said magazine, means for simultaneously driving a plurality of said cores held in said centers to wind thereon, and means for causing the said spaced centers of the several winding devices to approach in passing the magazine thereby to engage and remove from the magazine a new winding core.

37. A winding machine having therein a winding unit including a stop motion device normally tending to stop winding, means for normally locking the stop motion device in an ineffective position, a detector responsive to an abnormal condition of the unwinding strand, means for causing the winding unit and stop motion device to travel during winding, and means controlled by the detector for causing the traveling motion of the unit to release said locking means, so that the stop motion device can operate.

38. A winding machine having therein a plurality of devices for holding winding cores, means for moving said devices in a predetermined path in relation to the machine, means for simultaneously rotating cores held by a plurality of said holders to wind packages thereon, means actuable by the traveling motion of said holding devices to discharge the said packages wound on said cores from the respective holding devices, and means actuable by the traveling motion of said holding devices to replenish said holding devices with fresh cores.

39. A textile machine having therein a plurality of flexibly connected winding units each adapted to carry a winding package core, means for moving the units in a path of elongated shape including a straight portion of substantial length, means for rotating the several packages during their travel in said straight portion of path, and means operative during the winding to distribute the strand in conical layers of less length than the finished package length, and means for shifting the ranges of strand traverse axially of the packages.

40. A textile machine having therein a plurality of flexibly connected winding units each adapted to carry a winding package core, means for moving the units in a path of elongated shape including a straight portion of substantial length, means for rotating the several winding packages of the units from their cores during travel in said straight portion, means operative during such rotation to distribute strands on the packages in layers of less length than the finished package length, and means for shifting the ranges of strand traverse axially of the packages.

41. A textile machine having therein a plurality of flexibly connected winding units, means for moving the units in a path of elongated shape including a straight portion of substantial length, means for rotating the winding packages of the units during their travel along the said straight portion of the path, means associated with the respective units for distributing the strand on the packages in layers of less length than the finished package length, and means for shifting the ranges of strand traverse axially of the packages to wind full packages, the several units including means adapted to receive power during travel in said straight portion to rotate the package and operate the strand distributing means.

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DISCLAIMER

2,160,810.—*Edward J. Abbott*, Wilton, N. H. WINDING MACHINE. Patent dated June 6, 1939. Disclaimer filed June 19, 1942, by the assignee, *Abbott Machine Company*.

Hereby enters this disclaimer to claims 3, 4, 5, 6, 7, 8, 10, 11, 12, 13, 15, 16, 17, 18, 19, and 26 of said patent.

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DISCLAIMER

2,160,810.—*Edward J. Abbott*, Wilton, N. H. WINDING MACHINE. Patent dated June 6, 1939. Disclaimer filed September 20, 1943, by the assignee, *Abbott Machine Company*.

Hereby enters this disclaimer to claims 14 and 28 of said patent.

[*Official Gazette October 19, 1943.*]