July 27, 1954 W. E. MYERS 2,684,544

GAS HEATED ROLL TYPE IRONER

Filed Jan. 5, 1951 3 Sheets-Sheet 1

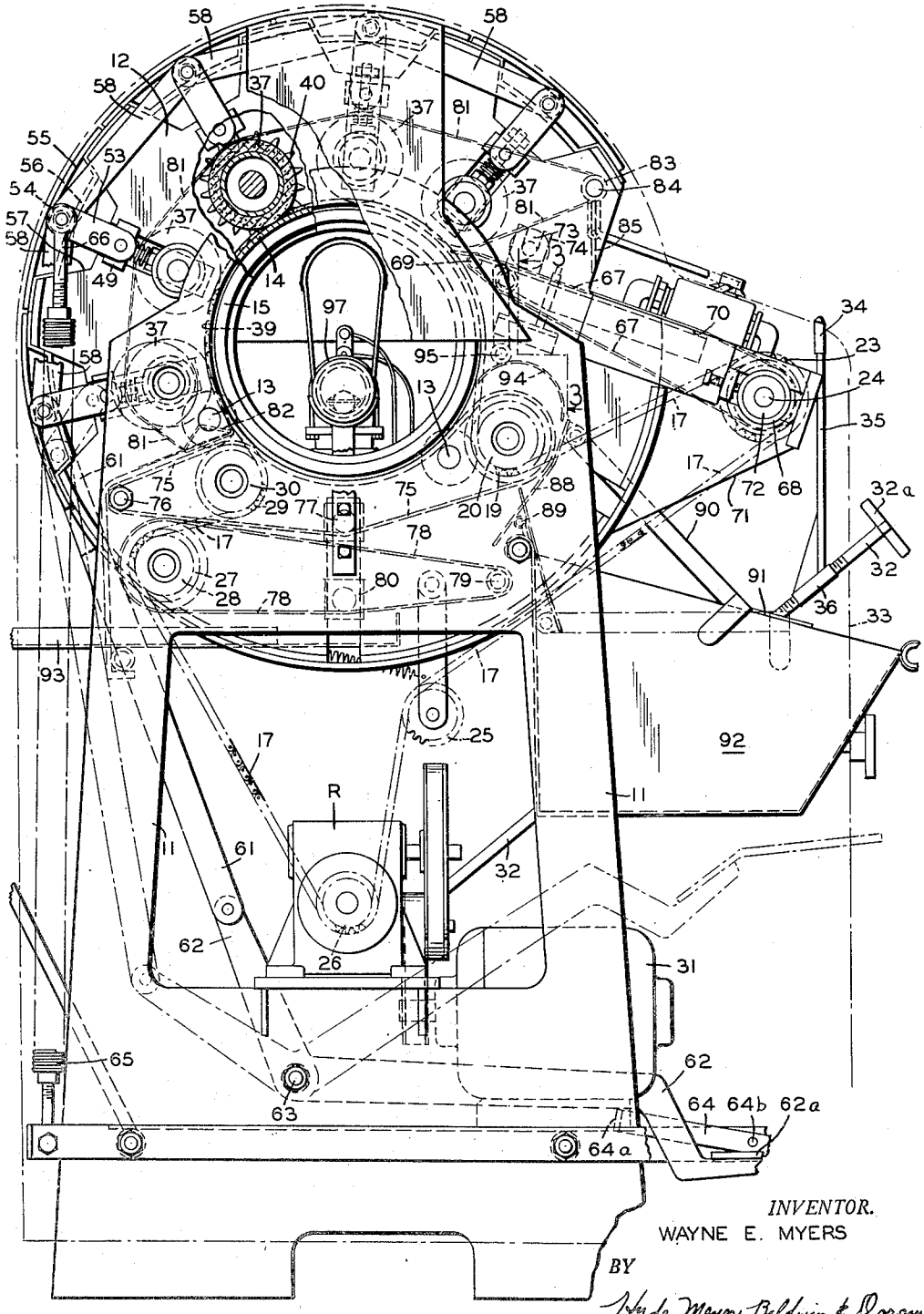


FIG.-1

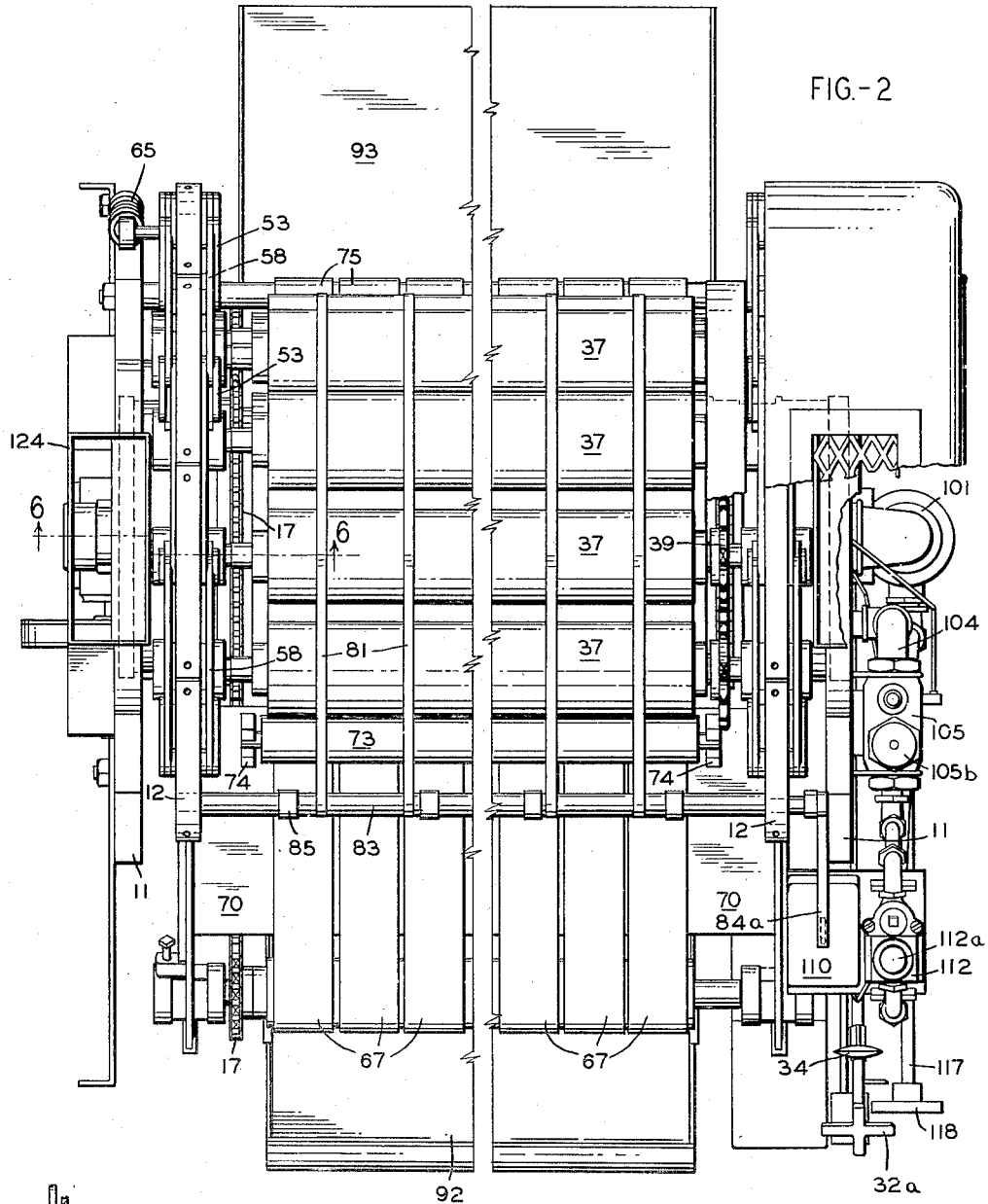
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GAS HEATED ROLL TYPE IRONER

Filed Jan. 5, 1951 3 Sheets-Sheet 2

FIG.-2



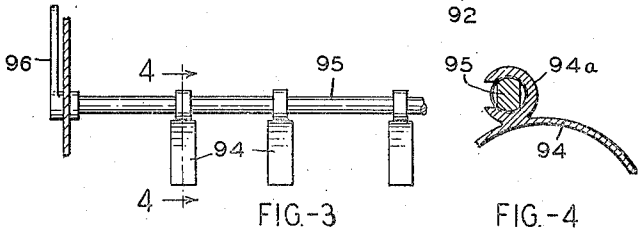


FIG.-3

FIG.-4

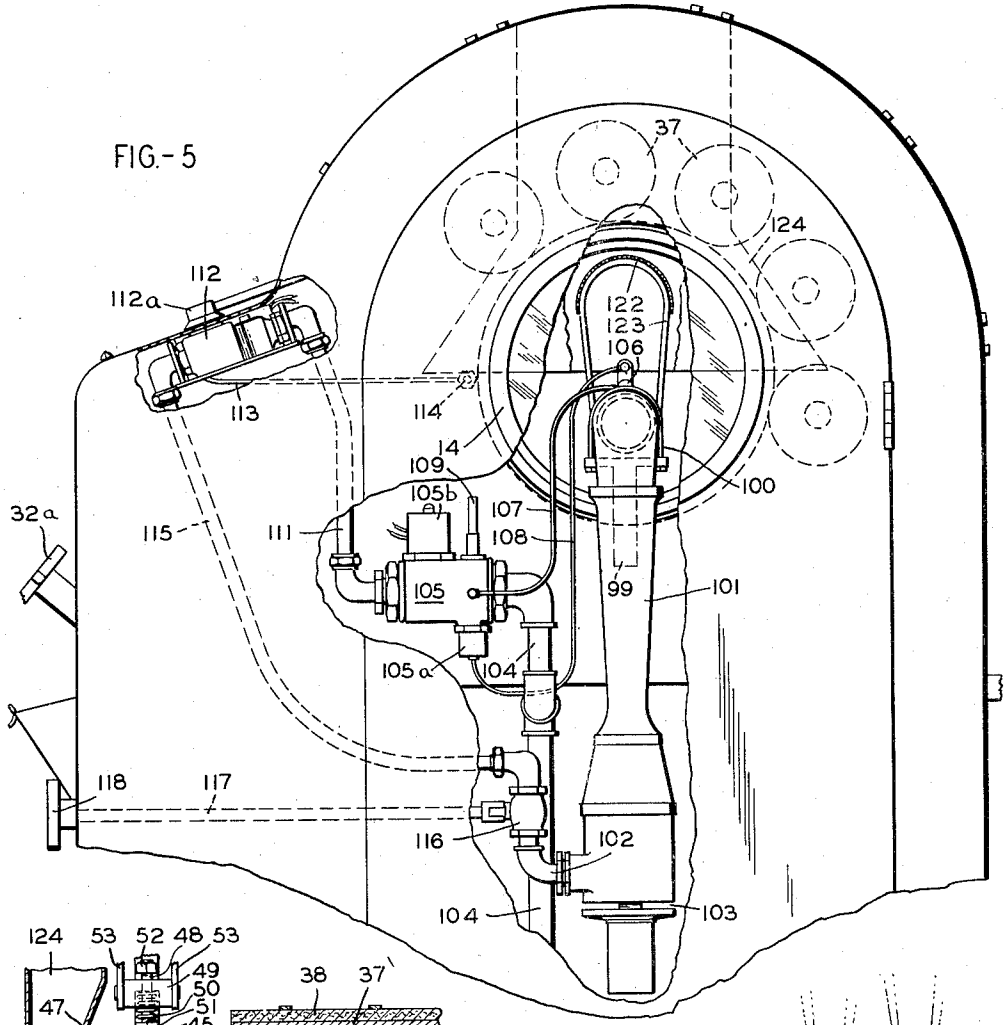
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GAS HEATED ROLL TYPE IRONER

Filed Jan. 5, 1951 3 Sheets-Sheet 3

FIG.-5



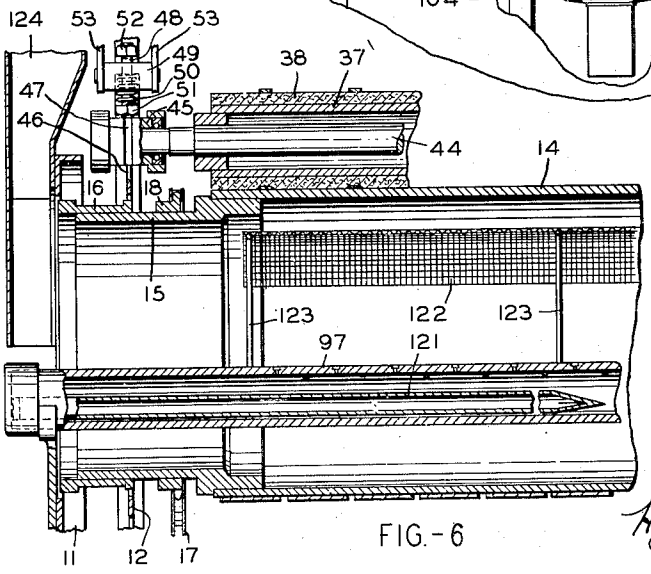


FIG.-6

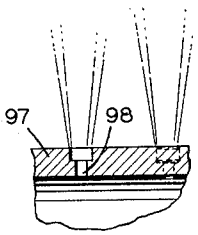


FIG.-7

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UNITED STATES PATENT OFFICE

2,684,544

GAS HEATED ROLL TYPE IRONER

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Application January 5, 1951, Serial No. 204,635

19 Claims. (Cl. 38—49)

1

The invention relates to a novel and improved power driven ironer of the type wherein the work to be ironed is advanced around a centrally located heated roll, and between it and a plurality of cooperating pressure rolls, usually having padding thereon, and biased into contact with the heated roll.

An object of the invention is to provide novel and improved means for controlling the application and relaxation of the pressure of the outer rolls against the heated roll. For convenience of reference, and avoidance of confusion, the inner heated roll will be hereinafter termed the "main roll," and the outer padded rolls will be termed the "ironing rolls."

A further object of the invention is to provide novel and improved burner equalizer means, disposable within the main roll, for evenly and efficiently heating said main roll, so as to avoid localized hot points.

A further object of the invention is to provide fluid fuel burner means in combination with said main roll, wherein adequate capacity is provided for supplying secondary air to the burner.

A further object of the invention is to provide fluid fuel burner means in combination with the main roll, the construction being adapted to prevent access of flame flashbacks to the padding.

A further object of the invention is to provide roll heating means as defined in the three immediately preceding paragraphs wherein the primary air supply requires no blower.

A further object of the invention is to provide novel and improved bypass deflector means, in combination with the main roll and the ironing rolls whereby the ironed work pieces issuing from the rolls may be deflected, at will, either towards the front or the rear of the machine.

A further object of the invention is to provide a bank of novel stripper fingers, which are individually readily removable, movable, or replaceable.

Other objects and advantages will be apparent from a study of the following specification, in conjunction with the accompanying drawings, wherein:

Fig. 1 is a left side elevational view, with housing parts broken away to reveal internal structure, showing a gas heated roll type ironer embodying the present invention.

Fig. 2 is a top plan view of the ironer.

Fig. 3 is a detail view of a fragmentary portion of the stripper assembly, taken approximately on the line 3—3 of Fig. 1.

Fig. 4 is a sectional view, somewhat enlarged, taken on the line 4—4 of Fig. 3.

2

Fig. 5 is a fragmentary right side elevational view, with the housing partially broken away to reveal burner, burner supply, and control parts.

Fig. 6 is a fragmentary vertical sectional view through the left end of the main roll, taken on the line 6—6 of Fig. 2.

Fig. 7 is an enlarged fragmentary sectional detail showing one of the burner orifices.

Referring to the drawings, the apparatus now to be described is supported on a spaced pair of end frames 11 which carry therewithin a spaced pair of circular end heads 12 (Fig. 2). Journaled in the end heads and end frames are two supporting idler rollers 13, one on each side, adapted to carry thereon the main roll 14. For this purpose the said main roll has attached to each of its open ends (Fig. 6) a shouldered sleeve 15 each of which has a fixed rabbetted fit in its respective roll end. Each said sleeve 15 has a sleeved extension 16, such extension at each roll end resting on a respective pair of the rollers 13 above mentioned.

The main roll is driven counterclockwise (as viewed in Fig. 1) by a roller link chain 17 which engages an annular sprocket 18 (Fig. 6) fixed on the outer periphery of sleeve 15. Chain 17 passes over the top of sprocket 18, under a sprocket 19 on a ribbon roll 20, around a sprocket 23 on a front ribbon feed roll 24, down over a slack take-up idler sprocket 25, then engages a drive sprocket 26, up over the sprocket 27 of another ribbon roll 28, and finally under another sprocket 29 freely rotatable on a ribbon roll 30 and thence back on sprocket 18.

The ribbon rolls just mentioned, as well as others cooperating therewith, drive various sets of ribbons for movement of the work, as will appear.

The chain drive sprocket 26 is driven from motor 31 through suitable controllable reduction gearing generally indicated by the letter "R" in Fig. 1, and such gearing is conventional. Variation in speed is achieved by manipulation of a shaft 32 having a handle 32a located to extend through a housing 33, and having in conjunction therewith an indicator 34 carried on a rod 35 secured to a sleeve 36 threaded on shaft 32. Rotation of shaft 32 moves the sleeve 36 on the shaft, so as to cause indicator 34 to correspondingly move along scale means (not shown) at the slot in the housing through which the rod extends. The scale has suitable speed indicating markings thereon for the operator's guidance.

As previously indicated, ironing of the work piece is effected by introducing the piece between the main roll 14 and one or more (preferably a

plurality) ironing rolls 37, five such rolls being shown distributed around and generally above a horizontal diametrical plane through the main roll axis. These ironing rolls have padding 38 thereon, and they are driven by means of a roller chain 39 secured around a sleeve on the right hand end of the main roll, there being a sprocket 40 on each ironing roll (Fig. 1) which engages said chain 39.

Since, as will appear, the ironing rolls are movable a limited distance towards and from the main roll, the teeth of sprockets 40 are made long enough to maintain driving connection with roller chain 39 during such movement.

All ironing rolls are similarly constructed and controlled, so description of one will suffice for all. The ironing rolls 37 are supported on shafts 44 journaled in bearings 45 slidable radially in slots 46 in circular end heads 12, the bearings having guide slots 47 which ride on the slot edges (Fig. 6). A threaded stud 48 is secured to each bearing, the stud being slidable through a block 49, there being a compression spring 50 between the block and an adjustable nut 51 on a threaded portion of the stud so as to adjust spring pressure. An upper nut 52 provides an end stop for block movement under spring bias, when inward pressure on the block (as will be described) is released. The block 49 is pivotally carried between spaced links 53, hereinafter termed pressure links. At their outer ends the links carry thereebetween a roller 54 which is circumferentially movable through a limited arcuate track at the periphery of end heads 12. The track is formed by outer parallel arcuate strips 55 and 56 which are secured to the periphery of the end heads. The end heads are cut out at 57 (Fig. 1) for this purpose.

The several shafts which support the respective rollers 54 are interconnected by links 58 which permit the several pressure link units to be operated simultaneously to apply or relieve pressure between the ironing rolls and the main roll. For this purpose the rearmost pressure link is connected to one arm 61 of a toggle (Fig. 1), the other arm 62 being of bell crank shape, and being fixed at its elbow on a cross rod 63 pivotally carried between the end frames 11. It will be apparent that downward foot pressure applied to the pedal portion 62*a* of arm 62 will move the toggle linkage from the broken line position to the full line position of Fig. 1, thereby also moving the pressure links 53 clockwise from the broken line to the full line position, as shown by the lowest one on the left. In their full line position the links are applying maximum inward pressure radially on the ironing rolls. In such pressure position the toggle 61, 62 is maintainable in over-center position so that the operator can remove the foot from pedal 62*a*. The toggle is broken by another foot operated lever 64 which has an operating tip 64*a* which can be pressed upwardly against the under side of arm 62. Lever 64 is pivoted at 64*b* on the frame. When the toggle is thus broken a spring 65 returns the pressure links 53 to pressure-relaxed position, the lower end of the spring being connected to the frame, and the upper end to an adjustable link 66 on the second nearest roller 54.

It should be mentioned that there are duplicate pressure control linkages 53 on both ends of the ironing rolls, as well as duplicate toggles 61, 62, the shaft 63 being the means whereby operation of the foot pedal 62*a* operates the toggles and responsive elements on both sides. Even pressure at both ends is thereafter secured by making suitable adjustments of the nuts 51 which compress the springs 50 in the pressure link. The structure and arrangement of the pressure links, connecting links, and associated parts is such that a combined camming and toggle action is secured in applying pressure to the ironing rolls.

As previously indicated, chain 17 provides the motive force for all rotating elements including the work-advancing ribbons. The feed ribbons 67 are of endless character, as is usual, rotating around driving roll 68 at the front (right side of Fig. 1) and around a rear rod 69 adjacent the main roll 14. The upper stretch of ribbons 67 is slidably supported on a feed table 70, which is supported by the circular end heads 12, and by frame extensions 71. The roll 68 is adjustably supported by bearings 72 in slots in the frame extension 71. A smoothing or drag roll 73 rides on ribbons 67 being generally upwardly displaceable in slots in brackets 74 at opposed sides of table 70.

A set of belts 75 is supported by rolls 20, 39, and a small rear tube or roll 76, there being a slack take-up roller 77 resting in vertically movable position on the lower stretch of belts 75. The weight of roller 77 pulls the upper stretch of belts 75 into working contact with the lower periphery of main roll 14. Below belts 75 is another set of belts 78 supported by roll 23 and a smaller roll, tube, or rod 79, there being a take-up roller 80 riding on the top of the lower stretch of belt 78. The various rolls mentioned in connection with the operation of the ribbons are supported on the circular end heads. A set of tapes 81 runs between the ironing rolls 37 and the main roll 14, to prevent the work from curling up around the ironing rolls during the ironing operation. Tapes 81 pass under a cross member 82 at the rear and around a tube or roll 83 (Fig. 2) at the front, as well as over the tops of the ironing rolls. The roll 83 is mounted on a cross rod 84 extending between the end heads 12, which rod also carries the usual safety finger guard 85.

Path selector means is provided for optional routing of the work either towards the operator at the front, or towards a possible operator at the rear.

Referring to Fig. 1, the path selector comprises a transversely extending plate 88, pivotally supported at 89 and having pivoted thereto, at each side edge of the plate, an arm 90 which is adapted to be moved from the full line position to the broken line position of Fig. 1. For stop purposes the arm 90 has an end portion which engages behind an abutment 91 on the bin or basket 92, as indicated in the full line position. In this position the workpiece travel is over the feed ribbon 67, around the top of main roll 14 and between the main roll and the ironing rolls 37, between the belts 75 and the main roll, over roll 20, onto guide plate 88, down onto delivery ribbons 78, and rearwardly to a rear work-receiving table 93.

If the deflector plate is dropped to its lower position, the work proceeds as before until it feeds over ribbons 75 on roll 20, whereupon plate 88 deflects the work directly into bin 92.

Stripper means is provided (Figs. 1, 3, and 4) for preventing adherence of the work to main roll 14. The provision of a stripper is not new, but, as far as is known to this inventor, the construction and support of the present stripper allows quick, easy, individual removal of any of the stripper fingers 94 in the event that the work

becomes entangled in one or more of the strippers. Normally the entire stripper assembly must be removed. Each finger 94 is supported on a rod 95 pivotally supported on end heads 12, there being a handle 96 on the rod for turning it when a finger is to be removed. As best seen in Fig. 4, rod 95 has flattened portions at each finger position, so as to reduce the rod in diameter in one direction. Each finger has a collar portion 94a slotted to admit the flattened portion of the rod. When in operating position, the relation of the stripper, collar, and rod is as shown in Fig. 4. To remove a finger the rod 95 is turned through 90 degrees whereupon the flattened rod portion is aligned with the slot in collar 94a so as to permit forward withdrawal of the finger.

The main roll is heated internally by a gas burner which is supplied with gas at one end. The burner consists of a pipe 97 (Fig. 6) having a plurality of rows of orifices 98 in its upper wall, there being three rows in the present case. These orifices are counter-bored (Fig. 7) which reduces the likelihood that the flames, which are upwardly directed, will be blown away. The burner pipe is supported at each end by a bracket 99, and at one end by a securing U-bolt 100 (Figs. 5 and 7). The ends of the roll 14 are open to furnish a plenteous supply of secondary air so that no blower is required. The burner pipe is capped on its left end, and at its right end is in communication with a standard mixing device 101 to which the gas enters at 102 and the primary air at 103. The gas supply comes from the main through pipe 104 and to a dual valve housing 105, which has two valves in series. The first valve is open only when a pilot burner 106 is lit, being controlled by a thermoresponsive element energized by the lighted burner. Pilot fuel is admitted through tube 107, and expansive fluid pressure is effective through tube 108 on a bellows in the bay 105a so as to open the first valve. To start the pilot a plunger button 109 is depressed so as to hold said first valve open and permit fuel to flow to the pilot burner which can then be lighted whereupon the first valve remains open and the plunger 109 can be released so that it automatically retracts.

The second valve in housing 105 is electrically controlled by a solenoid in bay 105b, said solenoid being responsive to a switch 110 (Fig. 2) which, when energized, operates the solenoid to open the switch. The switch is operated by a lever 84a associated with the shaft of the previously mentioned finger guard assembly, so that this second valve is open only when the finger guard is in protective position, the guard thereby serving as an element in manual control.

The finger guard is accordingly first moved to safe position, so as to energize the solenoid for opening the second valve, the first valve having previously been opened as already described, and gas can now flow from pipe 104 through the dual valve assembly to pipe 111 to a standard thermostat 112 having a temperature selector and control knob 112a. The thermostat is in communication through a capillary tube 113 with a temperature responsive bulb 114 which is in contact with roll 14. From the thermostat the gas travels through pipe 115 to a valve 116 having a control shaft 117 operated by a manually operated knob 118. When this valve is opened the gas can enter the mixing chamber 101 through pipe 102. Valve 116 permits the fuel to be cut off to the main burner without shutting off the motor which in the present embodiment is also controlled by switch 110 so that the main roll may run unheated for any desired interval. A manual valve is usually provided in the pilot line 107, but is not here shown.

The flame equalizer for the burner will now be described. Extending through the burner pipe 97 from the capped end is a pipe 121 which is flattened or tapered towards the incoming gas. This pipe 121 extends through only a fractional portion of the burner pipe, in this case approximately one-third the length of the burner. Certain adjustments can be made in both the relative length and diameter of the inner pipe 121 depending mainly on the length of the burner pipe 97, the gas pressure, etc. In an atmospheric-type burner such as that here shown, a dead-end or capped burner gives higher jets of flame near the capped end and I have found that a device such as pipe 121 serves to equalize the height of the jets along the length of the burner. A solid rod with tapered end can be substituted for pipe 121.

An arched metal screen or grid 122 is carried by rods 123 above the burner orifices and near the upper inner wall of main roll 14. When the burner is lighted this screen becomes red hot and prevents incomplete combustion which sometimes occurs as the fuel jets impinge on the inner roll surface.

In previous burner heated rolls of this type the ends of the roll, and especially the fuel supply end, are necked down or blocked to confine the flame, and as a result there was an inadequate supply of secondary air. Also in the prior construction it was customary to supply a blower in the primary air supply. By providing wide open main roll ends the present invention supplies adequate secondary air, and avoids an excess of primary air usually resulting from the use of a blower. The escape of combustion products through a flue 124 at both ends of the roll (one being shown in Fig. 3) draws in an adequate supply of secondary air at the roll's ends. The roll's ends are carried out to the side frames, thereby eliminating the possibility that flash backs might ignite padding 38, or cause other damage.

The present construction permits access of sixty to seventy per cent of secondary air, which supplements thirty to forty per cent of primary air fed to the mixing chamber which is highly effcient, and yet does not require a blower system. The lower end of each stack 124 is about even with the center of the main roll, which gives a good movement of cold air in at the bottom of the roll, the hot fumes escaping from the upper half of the roll.

The burner pipe may be positioned where desired, depending on the length of flame jet, etc., but I have found that in the embodiment shown the burner may be below the axial line of the main roll, giving a long, atmospheric type flame. This has been found to be better than the short, blue flame sometimes used, and the roll may be operated at a somewhat higher speed. It can be more easily thermostatically controlled without blow backs, and has a wider throttling range.

What I claim is:

1. A roll type ironer of the character described comprising a cylindrical, rotatable main roll, a plurality of ironing rolls spaced about a portion of the outer periphery of said main roll whereby to define an ironing zone wherein the work is advanced between said main roll and said ironing

rolls, a respective pressure applying device disposed outwardly beyond each ironing roll and effective inwardly against its respective ironing roll to apply or relieve working pressure between said ironing rolls and said main roll, said pressure applying device consisting of a pivoted toggle assembly, the inner toggle arm being effective against its ironing roll, and the outer end of the outer toggle arm being provided with a roller, an arcuate track fixed outwardly beyond said ironing rolls and beyond said toggle assembly, each said roller being movable in an arc along said track whereby to either break said toggle or apply inward lineal pressure on said roll by extending said toggle, and cross-connecting means operatively engaging the outer ends of all said pressure applying devices whereby to similarly operate each such pressure applying device simultaneously.

2. A roll type ironer of the character described comprising a cylindrical, rotatable main roll, a plurality of ironing rolls spaced about a portion of the outer periphery of said main roll, whereby to define an ironing zone wherein the work is advanced between said main roll and said ironing rolls, an arcuate track fixed radially outwardly beyond said ironing rolls, supporting means for each said ironing roll, each such means having an outer portion movable on said track into and out of radial alignment with a respective ironing roll and the main roll whereby respectively to increase and decrease working pressure between the last said ironing roll and said main roll, and means linking all said supporting means for simultaneous movement of all the ironing rolls for such increase and decrease of working pressure.

3. A roll type ironer of the character described comprising a hollow cylindrical rotatable main roll, a plurality of ironing rolls spaced about a portion of the outer periphery of said main roll to provide an ironing zone wherein the work is advanced between said ironing rolls and said main roll, means for urging each said ironing roll inwardly against said main roll comprising an arcuate track fixed radially outwardly beyond each said ironing roll, a plurality of pressure applying means having outer portions movable on said arcuate track and having inner portions effective against respective ironing rolls, cross linking means pivotally connected to the said outer portion of each such pressure applying means for simultaneously moving each such outer portion circumferentially along said arcuate track whereby to move each said pressure applying means into and out of radial registry with its respective ironing roll, so as to simultaneously apply or relax working pressure between said ironing rolls and said main roll.

4. A roll type ironer of the character described comprising a hollow cylindrical rotatable main roll, a plurality of ironing rolls spaced about a portion of the outer periphery of said main roll, elongated gas burner means extending along substantially the full length of the main roll and supported within said main roll, said main roll being open at both ends over its full diametrical extent whereby an adequate supply of secondary air is accessible to said burner through said full sized end opening.

5. An ironer as defined in claim 4 wherein said burner consists of a pipe having a plurality of openings along its upper wall surface, and pressure equalizing means within said pipe, and extending from the end of said pipe remote from the fuel supply end towards said fuel supply end, said pressure equalizing means being of elongated character and filling a part only of the internal area of the pipe.

6. An ironer as defined in claim 5 wherein said pressure equalizing means extends along the burner pipe for a minor fractional part of the length of said burner pipe, and is reduced in diameter towards the fuel supply end of said burner pipe.

7. An ironer of the character described comprising a cylindrical main roll, means for rotating said roll whereby to advance flatwork to be ironed thereon, means for applying ironing pressure to said work while it advances around said main roll, a plurality of stripping fingers adjacent said main roll and having tip portions in sliding contact therewith whereby to strip ironed work from said roll, and means for individually removably supporting each said finger in working position, comprising a cross bar generally parallel to and adjacent to the periphery of said main roll, said cross bar having spaced cylindrical portions flattened at diametrically opposed positions, each stripping finger having a projecting part with an open end slot therein adapted to slip over the aforesaid flattened portion so as to be readily removable when said flattened portion is in registry with said slot.

8. A roll type ironer of the character described comprising a cylindrical, rotatable main roll, a plurality of ironing rolls spaced about a portion of the outer periphery of said main roll whereby to define an ironing zone wherein the work is advanced between said main roll and said ironing rolls, a respective pressure applying device disposed outwardly beyond each ironing roll and effective inwardly against its respective ironing roll to apply or relieve working pressure between said ironing rolls and said main roll, cross connecting means operatively engaging the outer ends of all said pressure applying devices whereby to operate each such pressure applying device simultaneously, said main roll having a hollow interior throughout its axial length and being open at one end over its full diametrical extent, elongated gas burner means extending along substantially the full internal length of the main roll and being supported therewithin, whereby an adequate supply of secondary air is accessible to said burner through said full sized end opening.

9. An ironer as defined in claim 8 and wherein said gas burner means consists of a pipe having a plurality of openings along its upper wall surface, and pressure equalizing means within said pipe and extending within said pipe from the end of said pipe remote from the fuel supply end, said remote end being closed, said pressure equalizing means being of elongated character but of reduced cross sectional area, whereby to fill a part only of the internal area of the pipe.

10. An ironer as defined in claim 9 wherein said pressure equalizing means extends along the burner pipe for a minor fractional part of the length of said burner pipe, and is reduced in diameter towards the fuel supply end of said burner pipe.

11. An ironer as defined in claim 8 and comprising, in combination therewith, a plurality of stripper fingers adjacent said main roll and having tip portions in sliding contact therewith whereby to strip ironed work from said roll, and means for individually removably supporting each said finger in stripping position.

12. An ironer as defined in claim 8 and comprising, in combination therewith, a first set of

endless ribbons disposed below said main roll, and having their upper stretch in moving contact with said main roll whereby to receive the ironed work discharged downwardly from between the main roll and the ironing rolls, a second set of endless ribbons below said first set, means for moving said first set and said second set in opposite directions, and operator-operatable deflector means pivotally mounted adjacent the discharge point from said first set and adapted, by being preset to one or other of two positions, to deflect the ironed work discharged from the first set either onto or away from said second set.

13. A roll type ironer of the character described comprising a hollow cylindrical rotatable main roll, elongated gas burner means extending along substantially the full interior length of said main roll, and supported therein, said main roll being open at both ends over its full diametrical extent, and flue means in communication with said main roll interior for conveying the combustion products from said gas burner means whereupon secondary air is caused to flow unobstructedly inwardly through said open ends of said roll.

14. A roll type ironer as defined in claim 13, and including a plurality of ironing rolls spaced about a portion of the outer periphery of said main roll whereby to define an ironing zone wherein the work is advanced between said main roll and said ironing rolls, a respective pressure-applying device disposed outwardly beyond each ironing roll, and effective inwardly against its respective ironing roll to apply or relieve working pressure between said ironing rolls and said main roll, and cross-connecting means operatively engaging the outer ends of all said pressure applying devices whereby to similarly operate each such pressure-applying device simultaneously.

15. A roll type ironer as defined in claim 14 wherein the pressure-applying device consists of a pivoted toggle assembly, the inner toggle arm being effective radially against its ironing roll, and the outer arm of the toggle being swingable in an arc generally parallel to the periphery of said main roll, whereby to either extend or break the toggle, and wherein the cross-connecting means is pivotally connected to said outer end of said toggle arm.

16. A roll type ironer as defined in claim 15 wherein each outer toggle arm is provided at its outer end with a roller, and with an arcuate track fixed outwardly beyond said ironing rolls, and beyond said toggle assembly, each said roller being movable along said track whereby to either break said toggle or apply inward radial pressure on said roll by extending said toggle.

17. A roll type ironer as defined in claim 1 wherein each inner toggle arm is yieldably adjustably connected to its respective ironing roll whereby to adjust the working pressure between the ironing rolls and the main roll.

18. An ironer of the character described comprising a cylindrical main roll, means for rotating said roll whereby to advance flatwork to be ironed thereon, means for applying ironing pressure to said work while it advances around said main roll, a plurality of stripping fingers adjacent said main roll and having tip portions in sliding contact therewith whereby to strip ironed work from said roll, and means for individually removably supporting each said finger in working position comprising a cross bar generally parallel to and adjacent to the periphery of said main roll, said cross bar having spaced cylindrical portions flattened at diametrically opposed positions, each stripping finger having a projecting collar with a bored aperture therein slightly oversize with respect to said cylindrical portion, the wall of said aperture being slotted to slip over said flattened portion whereby, when relative rotation is produced between the cross bar and finger, the finger may be removed when said slotted wall portion is registered with said flattened portion.

19. An ironer of the character described comprising a cylindrical main roll, a plurality of ironing rolls disposed around an upper arcuate portion of said main roll, means for urging said ironing rolls into ironing engagement with said main roll, means for rotating said main roll whereby to cause work placed in contact with said main roll to pass between said main roll and said ironing rolls, a first set of endless ribbons disposed below said main roll, and having their upper stretch in moving contact with said main roll whereby to receive the ironed work discharged downwardly from between the main roll and the ironing rolls, a second set of endless ribbons below said first set, means for moving said first and said second set in opposite directions, and operator-operatable deflector means pivotally mounted adjacent the discharge point from said first set and adapted, by being preset to one or other of two positions, to deflect the ironed work discharged from the first set either onto or away from said second set, said ironer being provided with first and second work-receiving stations, the first work receiving station being adjacent and below the work discharge point of the first set of ribbons, the second work-receiving station being adjacent and below the work discharge point of the second set of ribbons, and wherein the deflector means is disposed in one working position to guide the work from the discharge point of the first set of ribbons towards the first work receiving station, and in its other working position to guide the work from the discharge point of the first set of ribbons onto the second set of ribbons and thence to the second work-receiving station, and operator-operatable means for swinging said deflector means to either one or the other of its two working positions.

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