



(51) International Patent Classification:

F02M 51/06 (2006.01) *F02M 61/16* (2006.01)
F02M 61/08 (2006.01) *F16J 3/04* (2006.01)
F02M 61/20 (2006.01)

(21) International Application Number:

PCT/EP2015/079752

(22) International Filing Date:

15 December 2015 (15.12.2015)

(25) Filing Language:

English

(26) Publication Language:

English

(30) Priority Data:

14199580.3 22 December 2014 (22.12.2014) EP

(71) Applicant: **CONTINENTAL AUTOMOTIVE GMBH**
[DE/DE]; Vahrenwalder Straße 9, 30165 Hannover (DE).

(72) Inventors: **FILIPPI, Stefano**; Via del Poderino 26, 57014 Castel' Anselmo Collesalveti (IT). **GRANDI, Mauro**; Via della Scopaia Nr. 36, 57128 Livorno (IT). **LENZI,**

Francesco; Via Muratori 29, 57128 Livorno (IT).

POLIDORI, Valerio; Via Grecia 35, 57128 Livorno (IT).

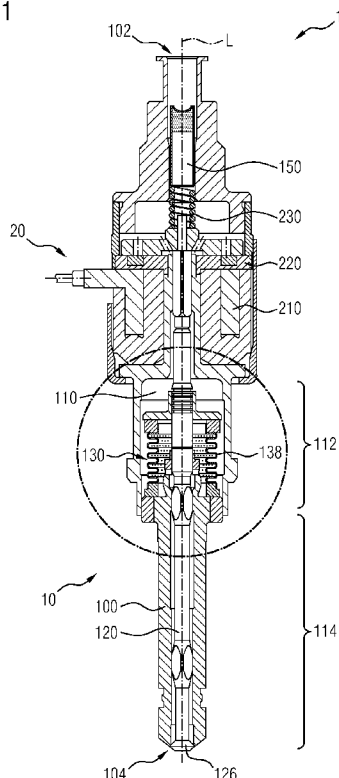
(81) **Designated States** (unless otherwise indicated, for every kind of national protection available): AE, AG, AL, AM, AO, AT, AU, AZ, BA, BB, BG, BH, BN, BR, BW, BY, BZ, CA, CH, CL, CN, CO, CR, CU, CZ, DE, DK, DM, DO, DZ, EC, EE, EG, ES, FI, GB, GD, GE, GH, GM, GT, HN, HR, HU, ID, IL, IN, IR, IS, JP, KE, KG, KN, KP, KR, KZ, LA, LC, LK, LR, LS, LU, LY, MA, MD, ME, MG, MK, MN, MW, MX, MY, MZ, NA, NG, NI, NO, NZ, OM, PA, PE, PG, PH, PL, PT, QA, RO, RS, RU, RW, SA, SC, SD, SE, SG, SK, SL, SM, ST, SV, SY, TH, TJ, TM, TN, TR, TT, TZ, UA, UG, US, UZ, VC, VN, ZA, ZM, ZW.

(84) **Designated States** (unless otherwise indicated, for every kind of regional protection available): ARIPO (BW, GH, GM, KE, LR, LS, MW, MZ, NA, RW, SD, SL, ST, SZ, TZ, UG, ZM, ZW), Eurasian (AM, AZ, BY, KG, KZ, RU, TJ, TM), European (AL, AT, BE, BG, CH, CY, CZ, DE, DK, EE, ES, FI, FR, GB, GR, HR, HU, IE, IS, IT, LT, LU, LV, MC, MK, MT, NL, NO, PL, PT, RO, RS, SE, SI, SK,

[Continued on next page]

(54) Title: VALVE ASSEMBLY AND FLUID INJECTION VALVE

FIG 1



(57) **Abstract:** A valve assembly (10) for a fluid injection valve (1) and a fluid injection valve (1) are disclosed. The valve assembly (10) comprises a valve body (100) having a cavity (110), a valve needle (120) which interacts with the valve body (100) for controlling fluid flow from a fluid outlet end (104) of the valve body (100), and a return spring element (130) which is preloaded for biasing the valve needle (120) towards a closing position. The return spring element (130) comprises a filter sieve (136) for filtering the fluid which flows through the cavity (110) to the fluid outlet end (104) in operation of the valve assembly (10).

SM, TR), OAPI (BF, BJ, CF, CG, CI, CM, GA, GN, GQ, **Published:**
GW, KM, ML, MR, NE, SN, TD, TG).

— *with international search report (Art. 21(3))*

Description

Valve assembly and fluid injection valve

- 5 The present disclosure relates to a valve assembly and to a fluid injection valve with the valve assembly.

Fluid injection valves are in widespread use, for example in automotive applications.

10

It is an object of the present disclosure to specify an improved valve assembly for a fluid injection valve.

15

This object is achieved by a valve assembly having the features of claim 1. Advantageous embodiments and developments of the valve assembly and the fluid injection valve are specified in the dependent claims, the following description and the drawings.

20

According to a first aspect, a valve assembly for a fluid injection valve is disclosed. According to a second aspect, a fluid injection valve comprising the valve assembly is disclosed. The fluid injection valve is in particular a fuel injection valve. It is preferably configured for injecting liquid fuel, such as gasoline, directly into a combustion chamber of an internal combustion engine.

25

30

The valve assembly comprises a valve body. The valve body has a fluid inlet end, a fluid outlet end, and a cavity which hydraulically connects the fluid inlet end to the fluid outlet end. Further, the valve assembly has a valve needle which interacts with the valve body for preventing fluid flow from the fluid outlet end in a closing position and which is displaceable away from the closing position to enable fluid flow out of the valve body from the fluid outlet end. In particular, a sealing element of the valve needle is in sealing mechanical contact with a valve seat of the valve body in the closing position and is axially displaceable along a longitudinal axis of the valve needle away from the valve seat for enabling the fluid flow.

35

In addition, the valve assembly comprises a return spring element which has a first end and a second end. The first and second ends are in particular axially opposite ends of the return spring element. The return spring element is seated against the valve
5 needle at the first end and against the valve body at the second end and is preloaded for biasing the valve needle towards the closing position. The return spring element comprises a filter sieve for filtering the fluid which flows through the cavity to the fluid outlet end in operation of the valve assembly. The
10 return spring element is in particular a one-pieced part or an assembly consisting of a plurality of inseparably joined components.

In this way, the valve assembly has a particularly low risk for
15 leakage at the fluid outlet end or for stuck-open type failures due to contamination with particles such as metal or plastic particles. Due to the integration of the filter into the return spring element, assembling of the valve assembly is particularly easy.

20 In one embodiment, the return spring element comprises an elastically deformable bellows which is pre-compressed in the valve assembly for biasing the valve needle towards the closing position. The bellows is, in other words, a corrugated sleeve.
25 The bellows is preferably a metal bellows. Adequate spring constants for biasing the valve needle towards the closing position are easily achievable by means of such a design.

In one development, the bellows comprises a multitude of filter
30 holes which perforate the bellows for forming the filter sieve or a section of the filter sieve. For example, the bellows has an inner circumferential surface and an outer circumferential surface and the filter holes penetrate the bellows from the outer circumferential surface to the inner circumferential surface, in
35 particular to enable a fluid flow in a generally radial direction from outside of the bellows towards a central axis of the bellows which is preferably parallel or coaxial to the longitudinal axis of the valve needle. By means of the bellows, a particularly large

filter area is achievable. In addition, the bellows - as a specific shape of a sleeve - allows for easy sealing of the return spring element against the valve body and/or the valve needle to define the fluid path through the filter sieve.

5

In one embodiment, the return spring element comprises a rigid, annular end piece at the first end and/or at the second end. The end piece or the end pieces are preferably in the shape of a ring or a sleeve. In an expedient embodiment, an axial end section or
10 axial end sections, respectively, of the bellows are fixed to the respective end piece(s) in fluid tight fashion. The end piece or the end pieces are preferably metal parts. They may expediently be connected to the bellows by means of a welded connection.

15 In one embodiment, the end piece or at least one of the end pieces comprises a multitude of filter holes which perforate the end piece for forming the filter sieve or a section of the filter sieve. For example, the end piece has a basic shape of a cylindrical shell which is perforated in radial direction by the
20 filter holes.

Preferably, the filter holes which penetrate the bellows and/or the end piece(s) have a mean diameter between 10 μm and 200 μm , preferably between 20 μm and 40 μm , the limits being included in
25 each case. For example the mean diameter of the filter holes is approx. 30 μm . In a preferred development, all filter holes have - apart from typical production tolerances - the same diameter. In this way, it is possible to filter particles of sizes which have a high risk for adversely affecting the sealing at the fluid
30 outlet end.

In one embodiment, the valve needle has a seat plate which comprises a spring seat of the valve needle. The first end of the return spring element sealingly rests against the spring seat of
35 the valve needle in an expedient development. In another embodiment, the valve body has a seat portion which comprises a spring seat of the valve body. The second end of the return spring element sealingly rests against the spring seat of the valve body

in an expedient development. In this way, the cavity of the valve body may expediently be divided in an upstream section which comprises the fluid inlet end and a downstream section which comprises the fluid outlet end, the two sections being separated
5 from one another by means of the return spring element and the region of the valve needle comprising the seat plate in such fashion that fluid has to pass the filter sieve of the return spring element on its way from the first section to the second section.

10 In one embodiment, the seat portion of the valve body is positioned, in direction along the longitudinal axis of the valve needle, between the seat plate and the sealing element of the valve needle which is positioned adjacent to the fluid outlet end. In this way, the filter sieve is positioned particularly close
15 to the fluid outlet end. In this way, it is operable to remove contamination inside a particularly large portion of the cavity. In contrast thereto, filters in conventional fluid injection valves are often placed adjacent to the fluid inlet end so that they are unable to remove particles which are, for example,
20 contaminating the cavity during assembling of the valve assembly.

In one embodiment, the seat plate, the seat portion and the return spring element extend circumferentially around the longitudinal axis. In this way, a particularly space-saving design and a
25 position of the filter sieve particularly close to the fluid outlet end are achievable.

In one embodiment, the cavity of the valve body has a first portion and a second portion downstream of the first portion. The second
30 portion is hydraulically connected to the first portion by means of at least one opening. The second portion extends from the seat portion towards the fluid outlet end and has a smaller cross-sectional area than the first portion. It is positioned such that it is laterally surrounded by the seat portion.

35

The valve needle extends from the second portion into the first portion so that the seat plate is positioned in the first portion. For example, a shaft of the valve needle is received in the second

portion and extends beyond the second portion into the first portion, where the seat plate is fixed to the shaft. At the downstream end of the first portion, the sealing element of the valve needle may project from the first portion. This is in particular the case when the fluid injection valve is an outward opening valve. In this type of valve, the valve needle is displaced away from the closing position in axial direction from the fluid inlet end towards the fluid outlet end for opening the valve.

Expediently, the valve needle - in particular the region which comprises the seat plate - together with the return spring element may cover the opening(s) in such fashion that fluid, on its way from the fluid inlet end to the fluid outlet end, has to pass the filter sieve before entering the second portion through the opening. In this way, particles can be filtered from the fluid in direct vicinity of the smaller cross-section second portion.

Further advantages, advantageous embodiments and developments of the valve assembly and the fluid injection valve will become apparent from the exemplary embodiments which are described below in association with the figures.

In the figures:

Figure 1 shows a longitudinal section view of a fluid injection valve according to a first exemplary embodiment,

Figure 2 shows an enlarged detail of the fluid injection valve of figure 1, and

Figure 3 shows a portion of the fluid injection valve according to a second exemplary embodiment in a longitudinal section view.

In the exemplary embodiments and figures, similar, identical or similarly acting elements are provided with the same reference symbols. In some figures, individual reference symbols may be omitted to improve the clarity of the figures.

Figure 1 shows a fluid injection valve 1 according to a first exemplary embodiment in a longitudinal section view with respect to a longitudinal axis L. The longitudinal axis L is, in the present embodiment, the central axis of a valve needle 120 and of a valve body 100 of a valve assembly 10 of the fluid injection valve 1. In addition to the valve assembly 10, the fluid injection valve 1 comprises and an electromagnetic actuator assembly 20.

10 The fluid injection valve 1 is in the present embodiment a fuel injection valve which is configured for injecting fuel -such as gasoline - directly into a combustion chamber of an internal combustion engine.

15 The valve body 100 extends from a fluid inlet end 102 to a fluid outlet end 104. In the present embodiment, these are also a fluid inlet end of the fluid injection valve 1 and a fluid outlet end of the fluid injection valve 1, respectively. The valve body 100 has a cavity 110 which establishes a fluid path from the fluid inlet end 102 to the fluid outlet end 104 and, thus, hydraulically connects the fluid inlet end 102 and the fluid outlet end 104.

In the present embodiment, the valve body 100 is assembled from a plurality of parts. At the fluid inlet end 102, it has the fuel inlet tube for connecting the fluid injection valve 1 to a fuel rail. Downstream of the fuel inlet tube, a tubular piece of the valve body 100 shapes an armature chamber in which an armature 220 of the electromagnetic actuator assembly 20 is received. Downstream of the tubular piece which shapes the armature chamber, a further piece of the valve body 100 shapes a spring chamber for a return spring element 130 of the valve assembly. A valve needle 120 of the valve assembly extends in the cavity 100 from said further piece of the valve body 100 downstream into a nozzle body of the valve body 100. The nozzle body extends to the fluid outlet end 104 and has a valve seat at the fluid outlet end 104.

The valve needle 120 has a sealing element 126 at the fluid outlet end 14 which mechanically interacts with the valve seat of the nozzle body for opening and closing the valve. In particular, the sealing element 126 is in sealing contact with the valve seat in a closing position of the valve needle to prevent fluid flow out of the fluid outlet end 104 and is displaceable away in axial direction along the longitudinal axis from the closing position to enable a fluid flow out of the valve body 100 from the fluid outlet end 104.

In the present case of an outward opening fluid injection valve 1, the valve needle 120 is displaceable in the axial direction directed from the fluid inlet end 102 to the fluid outlet end 104 away from the closing position to open a gap between the valve seat and the sealing element 126. The sealing element 126 may axially project beyond the valve body 100 at the fluid outlet end 104.

The return spring element 130 is seated against the valve body 100 and against the seat plate 122 of the valve needle 120 and is preloaded in the valve assembly 10, so that it is operable to bias the valve needle 120 towards the closing position. In order to displace the valve needle 120 away from the closing position, the armature 220 of the electromagnetic actuator assembly 20 is mechanically coupled to the valve needle 120. The electromagnetic actuator assembly 20 further comprises a coil 210 for generating a magnetic field to effect a magnetic force on the armature 220. By means of the magnetic force, the armature is moved axially within the armature chamber of the valve body 100 and, by means of its mechanical coupling to the valve needle 120 transfers a force on the valve needle 120 to move the latter away from the closing position in axial direction against the bias of the return spring element 130.

The portion of the valve assembly 10 which comprises the return spring element 130 is roughly indicated in figure 1 with a circle and shown in more detail in figure 2.

The return spring element 130 comprises a bellows 138 which is an elastically deformable sheet-metal part. At a first axial end 132 of the return spring element 130, a first annular end piece 142 is mounted on the bellows 138. The first end piece 142 is a solid metallic ring which is shifted over an axial end section of the bellows 138. At a second axial end 134 of the return spring element 130, a second annular end piece 144 is mounted on the bellows 138. The second end piece 144 is also a solid metallic ring which has a step in its outer surface and is shifted into a further axial end section of the bellows 138 so that the further axial end section of the bellows 138 abuts the step of the second end piece 144. Fluid-tight welded connections are established between the first end piece 142 and the bellows 138 and between the second end piece 144 and the bellows 138.

By means of the above-mentioned further piece of the valve body 100 which shapes the return spring chamber and by means of the nozzle body of the valve body 100, first and second portions 112, 114 of the cavity 110 of the valve body 100 are shaped. The first portion 112 comprises the return spring chamber in the present embodiment. The second portion 114 extends from the first portion 112 towards the fluid outlet end 104. It has a smaller cross-sectional area than the first portion 112.

The second portion 114 is hydraulically connected to the first portion 112 by means of a plurality of lateral openings 118 in an upstream end section of the nozzle body which projects into the return spring chamber. The shaft of the valve needle 120 extends through a central axial aperture at the upstream end of the nozzle body and basically completely fills the aperture so that the latter may not significantly contribute to hydraulically coupling the first and second portions 112, 114.

A shaft of the valve needle 120 extends from the sealing element 126 through the second portion 114 and further into the first portion 112 where the seat plate 122 is positioned and fixed to the shaft. The seat plate 122 may expediently extend radially outwards from the shaft. The seat plate 122 comprises a spring

seat 124 of the valve needle 120. At a downstream end of the return spring chamber, the valve body 100 has a seat portion 116 which defines a radially extending surface of the cavity 110 that represents a spring seat 106 of the valve body 100.

5

The first end 132 - more specifically the first end piece 142 - of the return spring element 130 sealingly rests against the spring seat 124 of the valve needle 120. The second end 134 - more specifically the second end piece 144 - of the return spring element sealingly rests against the spring seat 106 of the valve body 100. That the first end 132 or the second end 134 sealingly rest against the respective spring seats 124 and 106, respectively, means in particular that a fluid tight connection is established at the interface between the seat plate 122 and the first end piece 142 and between the second end piece 144 and the seat portion 116, respectively.

The seat portion 116 is arranged axially between the seat plate 122 and the sealing element 126 so that the return spring element 130 extends circumferentially around the longitudinal axis L and, thus, around the shaft of the valve needle 120. Further, the seat portion 116 extends circumferentially around the longitudinal axis L and around the second portion 114 of the cavity 110. In addition, the seat plate 122 extends around the longitudinal axis L and around the shaft of the valve needle 120 so that the seat plate 122 and the shaft together completely overlap the return spring element 130 in top view along the longitudinal axis L. In this way, the valve needle 120, together with the return spring element 130, covers the openings 118 which hydraulically connect the first portion 112 of the cavity 110 to the second portion 114.

In order to enable fluid passing through the return spring element 130 so that it can reach the openings 118 on its way from the fluid inlet end 102 to the fluid outlet end 104 and can flow from the first portion 112 further into the second portion 114, the bellows 138 has a multitude of filter holes 140 which perforate the bellows 138 in radial direction to form a filter sieve 136. In the present embodiment, the filter holes are of equal size, having

a diameter of 30 μm , for example. Fluid coming from the fluid inlet end 102 flows to an outer circumferential surface of the bellows 138 and has to flow through the sieve 136 into the inside of the bellows 138 to reach the openings 118 and flow further through the second portion 114 of the cavity 110 of the valve body 100 to the fluid outlet end 104.

The fluid injection valve 1 additionally comprises a filter tube 150 adjacent to the fluid inlet end 102. The filter tube 150 additionally represents a calibration tube for adjusting the preload of a calibration spring 230. The calibration spring 230 biases the armature 220 in axial direction into contact with the valve needle 120. The filter sieve 136 represented by the filter holes 140 in the bellows 138 is operable to filter particles from the fuel which are in the cavity 100 downstream of the filter tube 150, for example due to contamination during assembly of the fuel injection valve 1.

Figure 3 shows a second exemplary embodiment of a fuel injection valve 1 in a longitudinal section view corresponding to that of figure 2. The fuel injection valve 1 of the second embodiment corresponds in general to that of the first embodiment.

However, the return spring element 113 of the valve assembly of the fluid injection valve 1 according to the second embodiment has a bellows 138 which is not perforated by filter holes 140. In other words, the outer circumferential surface of the bellows 138 represents a simply-connected topological space. Instead of the bellows 138, the second end piece 144 - which is an axially elongated metallic sleeve, i.e. a cylindrical shell, in the present embodiment - is perforated by the filter holes 142 form the filter sieve 136. In a further variant (not shown in the figures), the bellows 138 as well as the second end piece 144 and/or the first end piece 142 may be perforated by filter holes 140 to form the filter sieve 136.

The invention is not limited to specific embodiments by the description on basis of these exemplary embodiments. Rather, it

comprises any combination of elements of different embodiments. Moreover, the invention comprises any combination of claims and any combination of features disclosed by the claims.

Claims

1. Valve assembly (10) for a fluid injection valve (1) comprising
 - 5 - a valve body (100) having a fluid inlet end (102), a fluid outlet end (104), and a cavity (110) which hydraulically connects the fluid inlet end (102) to the fluid outlet end (104),
 - 10 - a valve needle (120) which interacts with the valve body (100) for preventing fluid flow from the fluid outlet end (104) in a closing position and which is displaceable away from the closing position to enable fluid flow out of the valve body (100) from the fluid outlet end (104), and
 - 15 - a return spring element (130) which is seated against the valve needle (120) at a first end (132) and against the valve body (100) at a second end (134) and is preloaded for biasing the valve needle (120) towards the closing position,
wherein the return spring element (130)
 - 20 - is a one-pieced part or an assembly consisting of a plurality of inseparably joined components and
 - comprises a filter sieve (136) for filtering the fluid which flows through the cavity (110) to the fluid outlet end (104) in operation of the valve assembly (10).
- 25 2. The valve assembly (10) according to the preceding claim, wherein the return spring element (130) comprises an elastically deformable bellows (138) which is pre-compressed in the valve assembly (10) for biasing the
30 valve needle (120) towards the closing position.
3. The valve assembly (10) according to the preceding claim, wherein the bellows (138) comprises a multitude of filter
35 holes (140) which perforate the bellows (138) for forming the filter sieve (136) or a section of the filter sieve (136).

4. The valve assembly (10) according to one of the preceding claims, wherein the return spring element (130) comprises a rigid, annular end piece (142, 144) at the first end (132) and/or at the second end (134).
- 5
5. The valve assembly (10) according to the preceding claim, wherein the end piece (144) comprises a multitude of filter holes (140) which perforate the end piece (144) for forming the filter sieve (136) or a section of the filter sieve
- 10 (136).
6. The valve assembly (10) according to claims 2 and 4, wherein an axial end section of the bellows (138) is fixed to the end piece (144) in fluid-tight fashion and the end piece
- 15 (144) comprises a multitude of filter holes (140) which perforate the end piece (144) for forming the filter sieve (136) or a section of the filter sieve (136).
7. The valve assembly (10) according to one of the preceding claims, wherein the valve needle (120) has a seat plate
- 20 (122) which comprises a spring seat (124) of the valve needle (120) against which the first end (132) of the return spring element (130) sealingly rests.
8. The valve assembly (10) according to one of the preceding claims, wherein the valve body (100) has a seat portion
- 25 (116) which comprises a spring seat (106) of the valve body (100) against which the second end (134) of the return spring element (130) sealingly rests.
- 30
9. The valve assembly (10) according to claim 7 and 8, wherein - the seat portion (116) is positioned, in direction along a longitudinal axis (L) of the valve needle (120), between the seat plate (122) and a sealing element (126) of the
- 35 valve needle (120) which is positioned adjacent to the fluid outlet end (104).

10. The valve assembly (10) according to the preceding claim, wherein the seat plate (122), the seat portion (116) and the return spring element (130) extend circumferentially around the longitudinal axis (L).

5

11. The valve assembly (10) according to the preceding claim, wherein

10

- the cavity (110) has a first portion (112) and a second portion (114) downstream of the first portion (112), the second portion (114) being hydraulically connected to the first portion (112) by means of at least one opening (118),

15

- the second portion (114) extends from the seat portion (116) towards the fluid outlet end (104) and has a smaller cross-sectional area than the first portion (112) so that it is laterally surrounded by the seat portion (116),

20

- the valve needle (120) extends from the second portion (114) into the first portion (112) so that the seat plate (122) is positioned in the first portion (112), and
- the valve needle (120) together with the return spring element (130) cover the opening (118) so that fluid, on its way from the fluid inlet end to the fluid outlet end, has to pass the filter sieve (136) before entering the second portion (114) through the opening (118).

25

12. Fluid injection valve (1) comprising a valve assembly (10) according to one of the preceding claims.

1/2

FIG 1

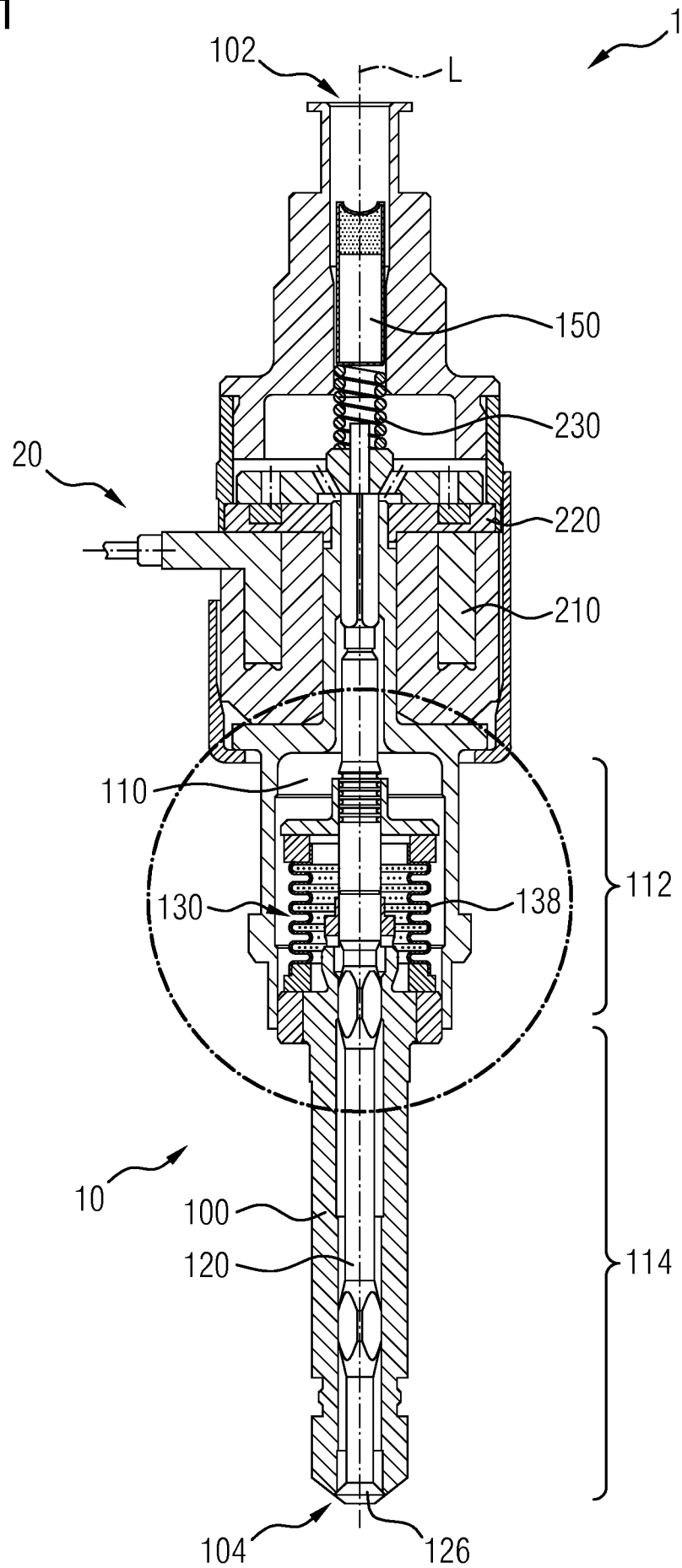


FIG 2

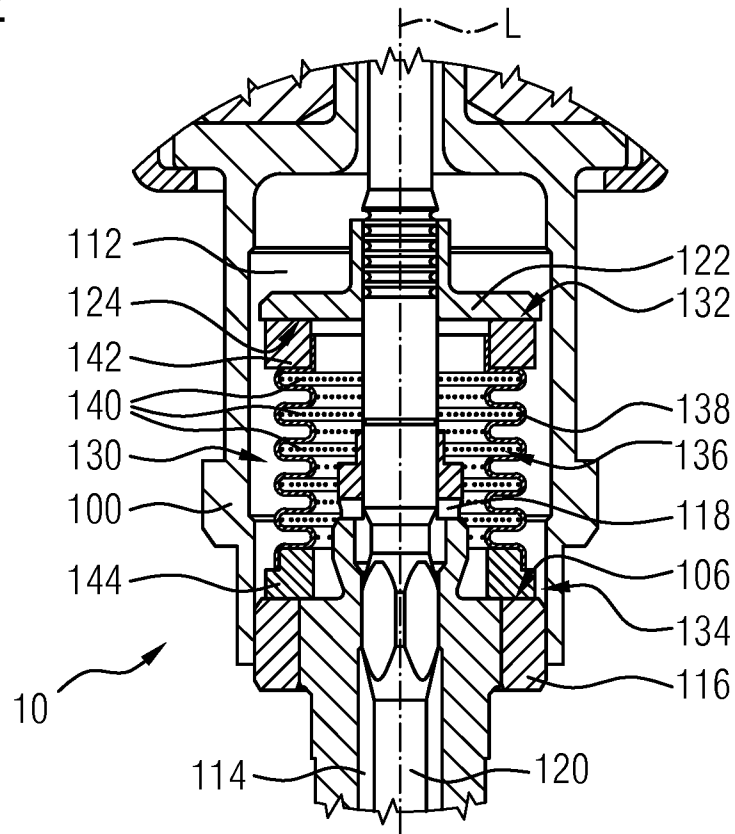
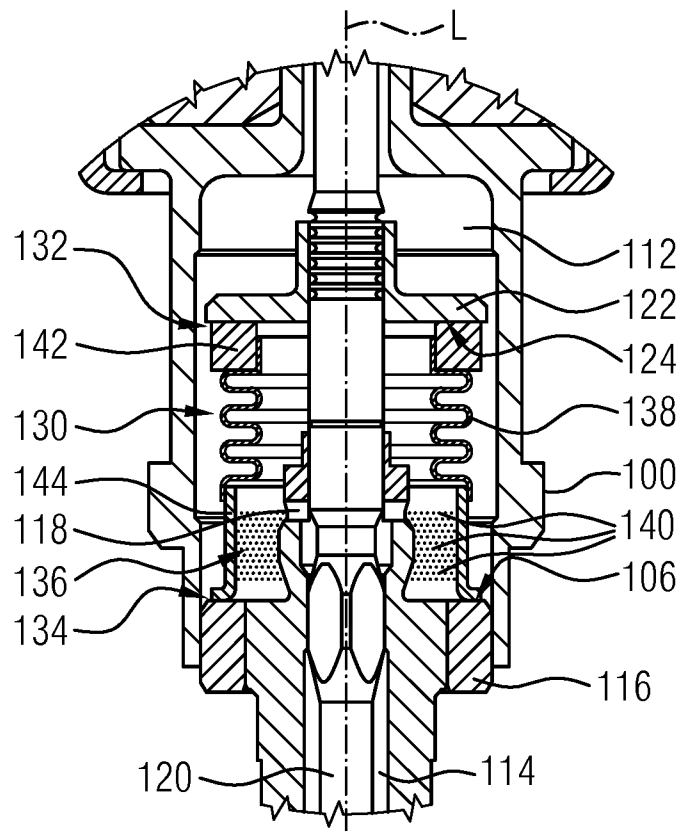


FIG 3



INTERNATIONAL SEARCH REPORT

International application No
PCT/EP2015/079752

A. CLASSIFICATION OF SUBJECT MATTER

INV. F02M51/06 F02M61/08 F02M61/20
ADD. F02M61/16 F16J3/04

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

F02M F16J

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

EP0-Internal, WPI Data

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	DE 10 2007 049963 A1 (BOSCH GMBH ROBERT [DE]) 23 April 2009 (2009-04-23)	1,4,5,7, 8,12
Y	figures	2,6

X	DE 41 31 535 A1 (BOSCH GMBH ROBERT [DE]) 25 March 1993 (1993-03-25)	1,4,5,7, 8,12
Y	figure 1	2,6

X	DE 10 2005 022672 A1 (BOSCH GMBH ROBERT [DE]) 23 November 2006 (2006-11-23)	1,12
Y	paragraphs [0019], [0024]; figure 1	2
	paragraph [0031]; figure 2	

X	DE 10 2009 027528 A1 (BOSCH GMBH ROBERT [DE]) 20 January 2011 (2011-01-20)	1,7-12
Y	paragraphs [0010], [0011], [0020], [0021]; figure 1	2

	-/--	



Further documents are listed in the continuation of Box C.



See patent family annex.

* Special categories of cited documents :

"A" document defining the general state of the art which is not considered to be of particular relevance

"E" earlier application or patent but published on or after the international filing date

"L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)

"O" document referring to an oral disclosure, use, exhibition or other means

"P" document published prior to the international filing date but later than the priority date claimed

"T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention

"X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone

"Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art

"&" document member of the same patent family

Date of the actual completion of the international search

19 February 2016

Date of mailing of the international search report

10/03/2016

Name and mailing address of the ISA/

European Patent Office, P.B. 5818 Patentlaan 2
NL - 2280 HV Rijswijk
Tel. (+31-70) 340-2040,
Fax: (+31-70) 340-3016

Authorized officer

Landriscina, V

INTERNATIONAL SEARCH REPORT

International application No
PCT/EP2015/079752

C(Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	EP 1 995 447 A1 (CONTINENTAL AUTOMOTIVE GMBH [DE]) 26 November 2008 (2008-11-26)	1,4,7-12
Y	paragraphs [0007] - [0009]; figures -----	2,6
Y	EP 1 731 754 A1 (SIEMENS AG [DE] CONTINENTAL AUTOMOTIVE GMBH [DE]) 13 December 2006 (2006-12-13) paragraph [0015]; figure 1 -----	2,6
A	US 2010/102146 A1 (KITAGAWA KAZUNORI [JP] ET AL) 29 April 2010 (2010-04-29) paragraph [0004]; figures -----	1,12

INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No

PCT/EP2015/079752

Patent document cited in search report	Publication date	Patent family member(s)	Publication date
DE 102007049963 A1	23-04-2009	DE 102007049963 A1	23-04-2009
		WO 2009053219 A1	30-04-2009
DE 4131535 A1	25-03-1993	DE 4131535 A1	25-03-1993
		DE 59201976 D1	24-05-1995
		EP 0558709 A1	08-09-1993
		JP 3093268 B2	03-10-2000
		JP H06502902 A	31-03-1994
		US 5340032 A	23-08-1994
		WO 9306359 A1	01-04-1993
DE 102005022672 A1	23-11-2006	NONE	
DE 102009027528 A1	20-01-2011	DE 102009027528 A1	20-01-2011
		EP 2290221 A1	02-03-2011
		JP 2011017340 A	27-01-2011
EP 1995447 A1	26-11-2008	NONE	
EP 1731754 A1	13-12-2006	DE 102005025952 A1	07-12-2006
		EP 1731754 A1	13-12-2006
US 2010102146 A1	29-04-2010	DE 102009021325 A1	12-05-2010
		JP 4628461 B2	09-02-2011
		JP 2010101267 A	06-05-2010
		US 2010102146 A1	29-04-2010