



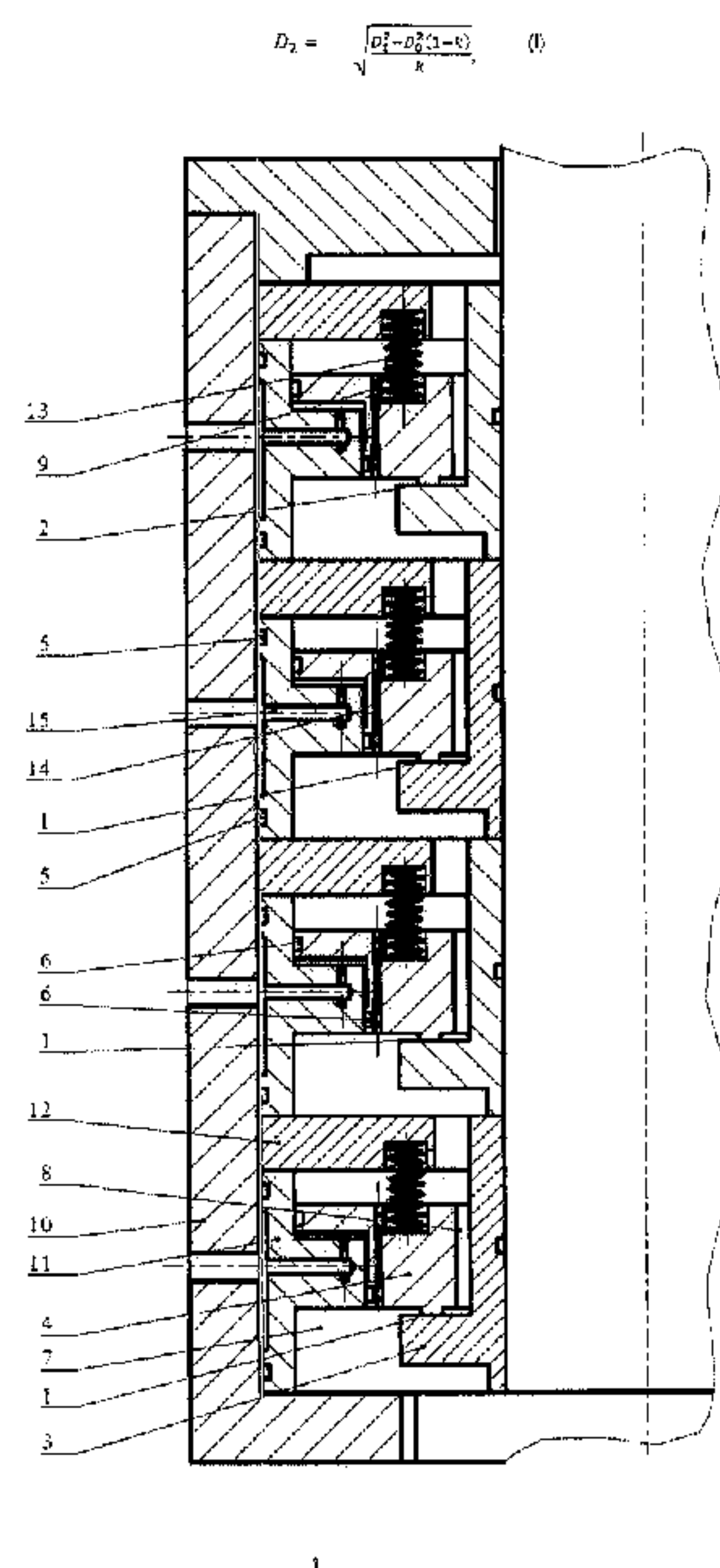
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(54) Titre : DISPOSITIF D'ETANCHEITE FACIALE
(54) Title: FACE SEAL



(57) **Abrégé/Abstract:**

The invention relates to end rotor seals installed in pump assemblies for the separation of media or pressure difference. The invention can be utilized in the design of pumps used in nuclear power stations, particularly main circulation pumps. The invention is aimed at increasing the reliability of the unit, as well as improving the manufacturability of components and the installation of the seal. In order to achieve the corresponding technical results, a multistage end seal design is proposed, comprising consecutively arranged working stages and an end stage, each of which comprises a rotor element arranged on a shaft, and a spring-loaded axially displaceable stator element connected to said rotor element and sealed with respect to the housing by means of sealing rings, wherein low- and high-pressure chambers of the working stages are consecutively connected by throttle openings in the stator element, which is configured so as to carry out the functions of an axially displaceable stepped piston which is sealed with respect to the housing. The outside diameters of the steps of the stator element piston are in the ratio (I), where D_2 is the diameter of the larger cylinder of the piston, and D_1 is the diameter of the smaller cylinder of the piston. This ratio ensures there is no additional force on the stator element when in normal operation, while, when the seal is open, allowing the action on the stator element to be increased by means of hydrostatic forces acting on the piston, said action being aimed at closing the seal.

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ABSTRACT

The invention relates to face seals of the rotors of pump units for separation of media or pressure drop. The invention can be used in the pumps used in nuclear power plants, in particular the main circulation pumps. The invention is aimed at improving the reliability of the assembly, as well as improving the workability of the parts and seal assembly. To obtain the appropriate technical results, a multistage face seal design is proposed, which comprises mounted in series work stages and end stage, each of which includes a rotor element located on the shaft and a spring-assisted axially movable stator element contacting it, sealed relatively to the casing with an O-ring, at the same time the cavities of high and low pressure of work stages are connected in series with restricted openings made in the stator element, which is designed so that to fulfill the functions of an axially movable step-like piston sealed relatively to the casing. The outer diameters of the stator element piston stages are made with the ratio $D_2 = \sqrt{\frac{D_1^2 - D_0^2(1-k)}{k}}$, where D_2 is the diameter of the larger piston cylinder, D_1 is the diameter of the smaller piston cylinder. This ratio ensures that there is no additional force on the stator element in the case of its normal operation, while allowing, in the case of seal opening, to increase the influence on the stator element aimed at seal closing due to the hydrostatic forces acting on the piston.

FACE SEAL

The invention relates to face seals of the rotors of pump units for separation of media or pressure drop.

5 The invention can be used in the pumps used in nuclear power plants, in particular the main circulation pumps.

A block of face seals is known (RF Patent No. 2084730 published on 20.07.1997, MPK F16J15/34). This design is a multistage face seal comprising mounted in series work stages and end stage, each of which includes a rotor element located on the shaft and a spring-assisted axially movable stator element contacting it, sealed relatively to the casing with an O-ring, at the same time the cavities of high and low pressure of work stages are connected in series with restricted openings in the stator casing. In the casing the step-like cylindrical borings are made in parallel to the shaft axis, communicating over the smaller diameter with high-pressure cavity, and over the larger one – with low-pressure cavity into which axially movable step-like cylindrical pistons are mounted, sealed in borings over larger and smaller diameters with the possibility of interaction of the smaller diameter piston end with a stator element. This design serves for forceful closing of each stage of the seal, if it is opened, by providing the unit with power step-like pistons in addition to the springs. The drawback of this design is presence of an additional mechanical assembly – the piston, which can affect performance of the product in case of its jamming, for example, in the lowest position. This can result in destruction of the seal friction pair. Also, in the mechanical seal design there is a design flaw associated with the piston operation. In connection with the fact that the piston is made step-like, when it operates, a cavity with a variable volume is formed between the end face restricted by the diameters of its larger and smaller cylinders, as well as reciprocal face surface in the body. Due to the complete tightness of the cavity, there will be created within it a vacuum or compression of the medium enclosed in the cavity. This effect will disturb free movement of the piston and, as a consequence, its correct operation.

Object-matter of the patent:

- increase in reliability of the face seal due to reduction of the number of slide assemblies;
- increase in workability due to simplification of the assembly and reduction of number of parts in the product.

In particular, the following technical results can be obtained, if the invention is implemented:

- decrease in time expenditures for maintenance of the face seal;
- decrease in metal consumption due to reduction of the number of parts.

As implementing of object to achieve the technical result, a face seal design is proposed, which differs from the prototype as follows.

Multistage face seal comprising mounted in series work stages and end stage, each of which includes a rotor element located on the shaft and a spring-assisted axially movable stator element contacting it, sealed relatively to the casing with an O-ring, at the same time the cavities of high and low pressure of work stages are connected in series with restricted openings made in the stator element. The casing, in turn, comprises an external cylinder, in which the step-like ring and thrust rings are successively mounted – by one per each step. Thrust ring serves to lock spring between it and the stator element. Stator element and step-like ring of the casing are designed so that the stator element fulfills functions of the axially movable step-like piston sealed with rubber seals, at the same time a portion of the piston with smaller diameter is located in the high pressure region – before seal, and the second portion with larger outer diameter – on the side of lower pressure or after seal. In the area of transition of a smaller cylinder to a larger one a lowering is performed on the stator element. In turn, in the step-like ring reciprocal to the stator element, in the area of transition of cylinder diameters reciprocal to the stator element piston a through hole is made, so that at any position of the stator element not to extend beyond the lowering boundaries on it. This hole serves for compensation for the vacuum-compression in the formed cavity volume, when the stator element moves. Hole seal in the casing is implemented between the outer cylinder and the step-like ring by two rubber rings, between which it is located.

The working function of the seal is as follows. At normal operation of the face seal unit, a part of the working pressure is activated at each working stage of the seal using restricted openings, depending on the throughput capacity of the restricted opening. As a result, the stator element piston is affected by the force $F_1 = P_1 \times \pi \frac{D_1^2 - D_0^2}{4}$ from the side before seal, where D_1 is the smaller diameter of the stator element piston, D_0 – the internal diameter of the stator element, P_1 is pressure before the seal, the force $F_2 = P_2 \times \pi \frac{D_2^2 - D_0^2}{4}$ acts on the stator element piston from side after the seal, where D_2 is the larger diameter of stator element piston, P_2 is pressure after seal equal to kP_1 , where k ($0 < k < 1$) is coefficient of pressure drop due to the restricted opening in the stator element. Forces F_1 and F_2 should be equal, so that the stator element in the operating position is affected by the spring force only $D_2 = \sqrt{\frac{D_1^2 - D_0^2(1-k)}{k}}$. If the seal is opened, pressure P_2 after the seal is equalized with pressure P_1 before the seal, it causes the situation that the force $F_2' = F_1 \times \frac{D_2^2 - D_0^2}{D_1^2 - D_0^2}$ starts to act on the stator element piston surface after the seal more than before the seal, which causes it to move towards the gap closing direction. In order to compensate for changes in volume in the piston chamber during its movements, the area formed by the lowering on stator element, the counterpart of the step-like ring and the through hole is connected with the expansion tank.

The invention is explained in the drawing, which demonstrates a general view (axial section) of the face seal.

Multistage face seal comprises the casing 10 made in the form of outer cylinder, which accommodates step-like rings 11 and thrust rings 12 mounted in series - by one per each stage, as well as

rotor elements 3 located on the shaft with contact surfaces 1 and spring-assisted axially movable stator elements 4 with contact surfaces 2 contacting with them.

Springs 13 are mounted between the thrust rings 12 and movable stator elements 4. They provide pressing force between the contact surfaces 1 and 2.

5 High pressure cavities 7 formed by the step-like rings 11, the stator elements 4 and the rotor elements 3 are connected with the low pressure cavities 8 formed between the thrust rings 12, the step-like rings 11, the stator elements 4 and the rotor elements 3, restricted openings 9 made in the stator elements 4.

The cavities of high and low pressure are separated by rubber rings 6.

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Stator element 4 and step-like ring 11 are designed so that the stator element fulfills functions of the axially movable step-like piston, at the same time a portion of the piston with smaller diameter is located in the high pressure region – before seal, and the second portion with larger outer diameter – on the side of lower pressure or after seal. In the area of transition of a smaller cylinder to a larger one a lowering 14 is performed on the stator element 4. In turn, in the step-like ring 11 reciprocal to the stator element 4, in the area of transition of cylinder diameters a through hole 15 is made, so that at any position of the stator element not to extend beyond the lowering boundaries on it. This opening serves to compensate for the vacuum-compression in the volume between the step-like ring 11 and the stator element 4 restricted by the rubber rings 6 of the cavity formed when the stator element 4 moves.

20 The seal of this volume between the outer cylinder 10 and the step-like ring 11 is made with two rubber rings 5.

The working function of the seal is as follows. At normal operation of the face seal unit, a part of the working pressure is activated at each working stage of the seal using restricted openings 9, depending on the throughput capacity of the restricted opening. As a result, the stator element 4 piston is affected by the force $F_1 = P_1 \times \pi \frac{D_1^2 - D_0^2}{4}$ from the side of high pressure cavity 7, where D_1 is the smaller diameter of the stator element piston, D_0 – the internal diameter of the stator element, P_1 is pressure from the side of high pressure cavity. From the low pressure cavity 8 side, the force $F_2 = P_2 \times \pi \frac{D_2^2 - D_0^2}{4}$ acts on the stator element piston, where D_2 is the larger diameter of stator element piston, P_2 is pressure from the side of low pressure cavity equal to kP_1 , where k ($0 < k < 1$) is coefficient of pressure drop due to the restricted opening in the stator element. Forces F_4 and F_{13} should be equal, so that the stator element 4 in the operating position is affected by the spring force 13 only $D_2 = \sqrt{\frac{D_1^2 - D_0^2(1-k)}{k}}$. If the seal is opened, pressure in the cavities is equalized, it causes the situation that the force $F_2' = F_1 \times \frac{D_2^2 - D_0^2}{D_1^2 - D_0^2}$ starts to act on the stator element piston surface, which causes it to move towards the gap closing direction.

35 In order to compensate for pressure changes in volume formed between the step-like ring 11, the stator element 4 and restricted by the rubber rings 6, formed during movements of the stator element 4, it is connected with the expansion tank beyond the face seal using the through hole 15.

CLAIMS

1. Face seal comprising mounted in series work stages, each of which includes a rotor element located on the shaft and a spring-assisted axially movable stator element contacting it, sealed relatively to the casing with an O-ring, at the same time the cavities of high and low pressure of work stages are
5 connected in series with restricted openings made in the stator element, characterized in that

the casing comprises outer cylinder, which accommodates step-like ring and thrust ring mounted in series - by one per each stage, wherein the step-like ring is sealed relatively to the cylinder and is reciprocal to the stator element, and the thrust ring acts as the spring lock between it and the stator element,

10 restricted openings are made in stator element, which is designed so that to fulfill functions of the axially movable step-like piston sealed relatively to the step-like ring of the casing, at the same time a portion of the piston with smaller diameter is located in the high pressure region – before the seal, and the second portion with larger outer diameter – on the side of lower pressure or after

seal, also in the casing outer cylinder and the step-like ring, in the region of transitions of the
15 cylinder diameters reciprocal to the stator element pistons a through hole is made, which connects the formed cavity between casing and stator element with the expansion tank.

2. The face seal according to claim 1, characterized in that the outer diameters of the stator element piston stages are made with the ratio $D_2 = \sqrt{\frac{D_1^2 - D_0^2(1-k)}{k}}$, where D_2 is the diameter of the larger piston cylinder, D_1 is the diameter of the smaller piston cylinder, D_0 is the internal diameter of the stator element;
20 k is the coefficient of pressure drop due to the restricted opening in the stator element.

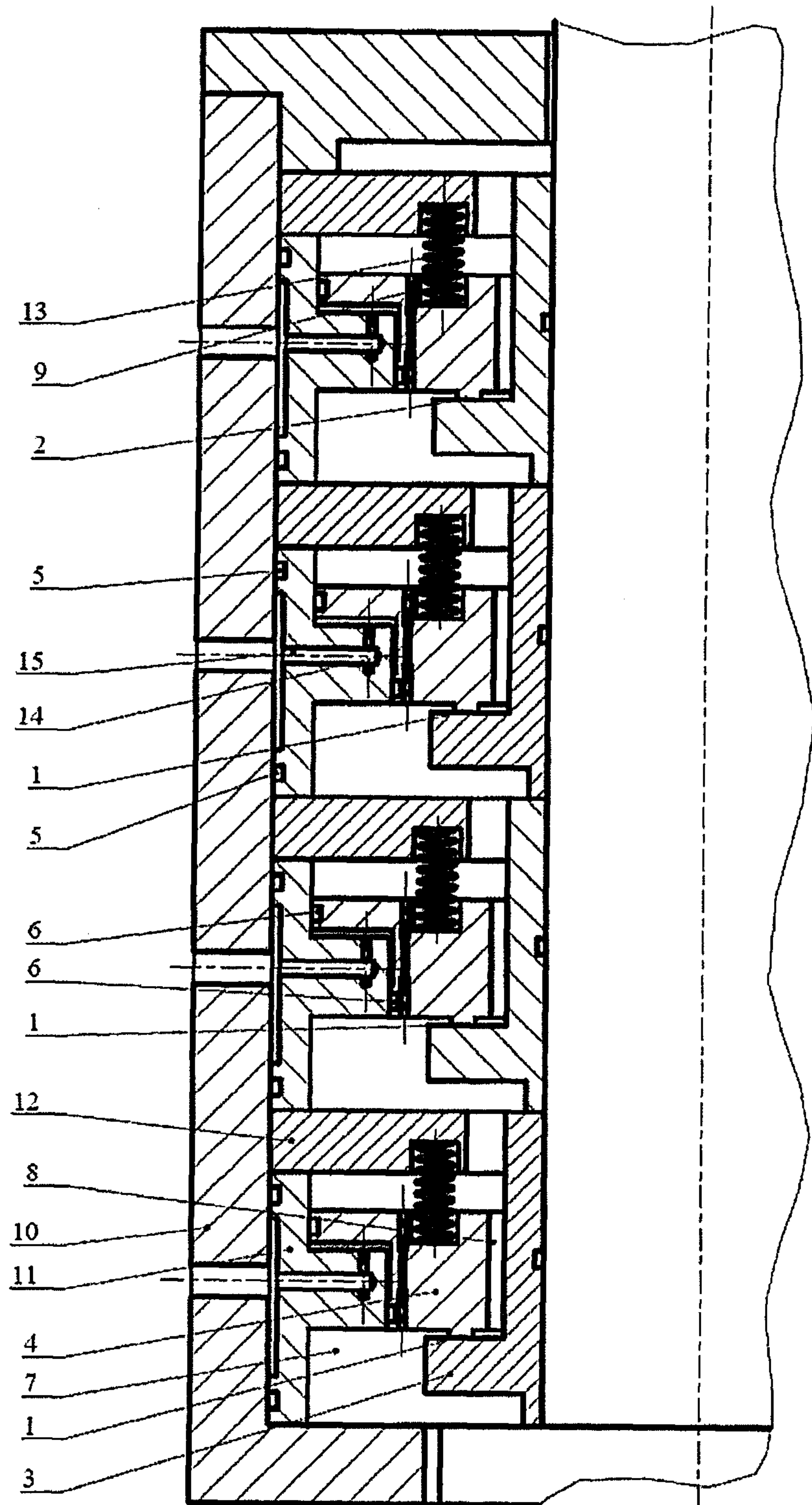
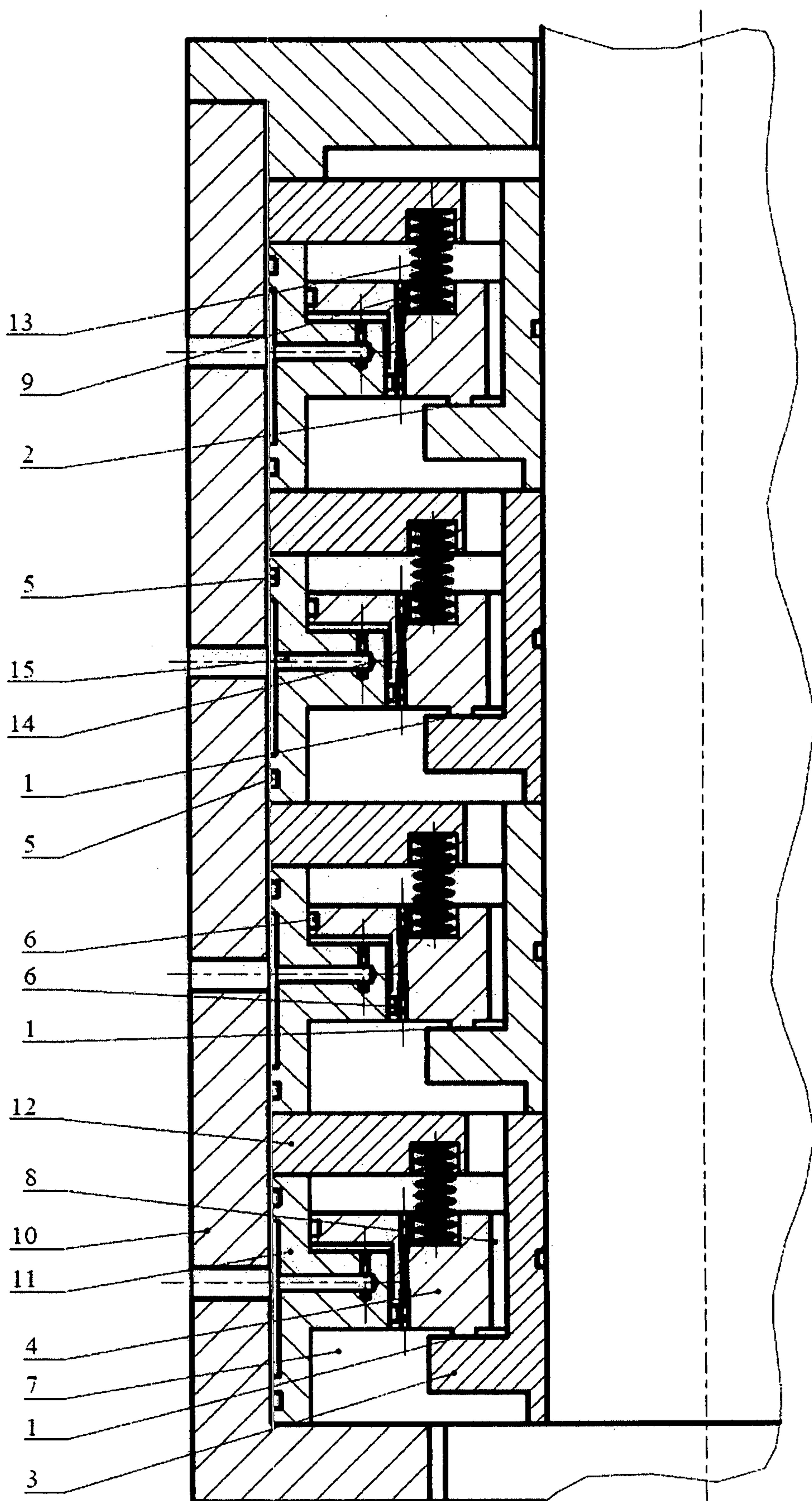


Fig. 1

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$$D_2 = \sqrt{\frac{D_1^2 - D_0^2(1-k)}{k}}, \quad (I)$$



Фиг. 1