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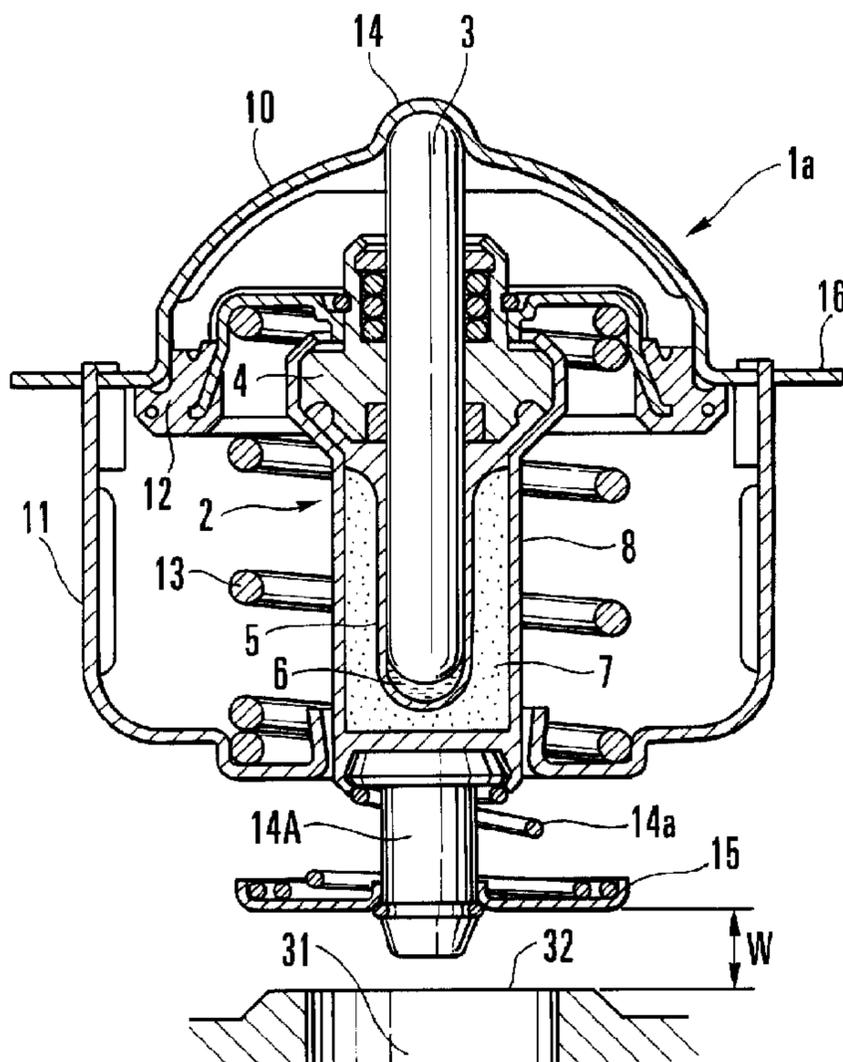
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(54) Titre : THERMOSTAT POUR UN SYSTEME DE REFROIDISSEMENT DE MOTEUR D'AUTOMOBILE

(54) Title: THERMOSTAT FOR AN AUTOMOTIVE ENGINE COOLING SYSTEM



(57) Abrégé/Abstract:

A thermostat has a housing having a flange, an actuating rod secured to the housing at a first end thereof, a guide member slidably mounted on the actuating rod, a resilient seal spool provided around a second end portion of the actuating rod and secured to the guide member, a heat sensitive cylinder housing the seal spool and secured to the guide member, wax pellets provided in the heat sensitive cylinder to enclose the seal spool. The flange has a hole so as to pass a coolant, and the thickness of the resilient seal spool being set between 25 % and 5 % of the diameter of the actuating rod, thereby reducing a spring constant of a return spring of the thermostat.

THERMOSTAT FOR AN AUTOMOTIVE ENGINECOOLING SYSTEMAbstract of the Disclosure

5           A thermostat has a housing having a flange, an  
actuating rod secured to the housing at a first end  
thereof, a guide member slidably mounted on the  
actuating rod, a resilient seal spool provided around a  
second end portion of the actuating rod and secured to  
10 the guide member, a heat sensitive cylinder housing the  
seal spool and secured to the guide member, wax pellets  
provided in the heat sensitive cylinder to enclose the  
seal spool. The flange has a hole so as to pass a  
coolant, and the thickness of the resilient seal spool  
15 being set between 25 % and 5 % of the diameter of the  
actuating rod, thereby reducing a spring constant of a  
return spring of the thermostat.

## TITLE OF THE INVENTION

## THERMOSTAT FOR AN AUTOMOTIVE ENGINE COOLING SYSTEM

## BACKGROUND OF THE INVENTION

The present invention relates to a wax type  
5 thermostat for controlling temperature of a coolant of  
an automotive engine.

Referring to Fig. 6 showing a conventional cooling  
system for an automotive engine, the system has a  
thermostat 1 which is disposed in an inlet side passage  
10 of water jackets 20.

The cooling system comprises a first coolant  
passage 24 disposed between an upper outlet 21 of the  
water jackets 20 and an upper inlet 23 of a radiator 22,  
and a second coolant passage 30 provided between a lower  
15 outlet 25 of the radiator 22 and a lower inlet 29 of the  
water jackets 20, including a thermostat cap 26, a  
thermostat housing 27 and a water pump 28. A bypass  
passage 31 is provided between a junction J of the first  
passage 24 and the thermostat housing 27 so as to  
20 communicate the first passage 24 with the second passage  
30 without passing the radiator 22. The thermostat 1 is  
hermetically secured to the housing 27 by the thermostat  
cap 26. The thermostat 1 has a main valve 12 for  
closing the second passage 30 and a bypass valve 15 for  
25 closing a bypass port 32 of the bypass passage 31.

In Fig. 6, the reference A' designates a measuring  
point for measuring the temperature of the coolant in

the housing 27, and B' designates a measuring point provided in the second passage 30 adjacent to the thermostat cap 26 and upstream of the thermostat 1 for measuring the temperature of the coolant in the second  
5 passage 30. The reference C designates a measuring point for measuring the flow rate of the coolant in the second passage 30. The reference numeral 33 designates a cooling fan.

The thermostat 1 is operated by a thermo-actuator.  
10 The thermo-actuator comprises an actuating steel rod and a resilient seal spool which is slidably engaged with the rod. The seal spool is inserted in a heat sensitive cylinder filled with wax pellets.

As shown in Fig. 7, a perforation 19a is formed in a  
15 flange 16 of the thermostat 1, and a jiggle valve mechanism 17 having a jiggle valve 18 is movably engaged in the perforation 19a.

During the operation of the engine, the jiggle  
valve 18 is closed by the pressure of the coolant in the  
20 second coolant passage 30 as shown in Fig. 7. When the engine stops, the jiggle valve opens. Thus, the coolant can be supplemented in the direction of the arrow.

During the cold engine state, the main valve 12 of  
the thermostat 1 is closed as shown in Fig. 6, and the  
25 jiggle valve 18 is also closed by the coolant pressure, while the bypass valve 15 integrated with the main valve 12 is fully opened. Thus, the coolant drawn from the outlet 21 of the water jackets 20 does not pass through

the radiator 22. The coolant is circulated by the water pump 28 through the junction J of the first passage 24, bypass passage 31, housing 27, and inlet 29 of the water jackets 20 as indicated by arrows. Thus, the  
5 temperature of the coolant in the housing 27 quickly rises.

However, since the coolant in the radiator 22 and the thermostat cap 26 is not circulated, the temperature rising rate of the coolant temperature B therein is  
10 slow. Therefore, as shown in a record of Fig. 8, after even if the temperature A at the point A' becomes 87°C which is an opening temperature of the main valve 12, the temperature B at the point B' is merely 45°C. There is a difference of 42°C between the temperatures A  
15 and B.

When the main valve 12 of the thermostat 1 opens, the coolant of a low temperature is drawn from the lower outlet 25 of the radiator 22 and fed to the thermostat housing 27 through the second passage 30. Consequently,  
20 the temperature B of the coolant at the point B' is further lowered by 13°C. As a result, the difference between the temperature B of the coolant in the passage 30 and the temperature A of the coolant in the housing 27 increases to 55°C. The area of the part shown by the  
25 hatching indicates energy loss in the period. It will be understood that the time of the abscissa indicates the elapse from the time at 60°C of the temperature A.

Since the heat sensitivity of the thermostat 1 is

low, the response of the thermostat delays with respect to the change of the coolant temperature. Therefore, the main valve 12 opens after the temperature has become considerably higher than the predetermined opening  
5 temperature 87°C. When the main valve 12 opens, the temperature of the coolant is lowered. The main valve 12 closes after the coolant temperature has considerably decreased lower than a predetermined closing  
10 temperature. Then, the coolant temperature rises. Namely, there is a large heat overshoot in control of the coolant temperature, so that the main valve is repeatedly opened and closed. When the main valve 12 closes, a surge pressure occurs at the upstream of the main valve.

15 The heat overshoot causes cracks of the cylinder block and cylinder head of the engine, and the surge pressure causes breakdown of the thermostat 1 and the radiator 22.

Since, mentioned in above, the jiggle valve  
20 mechanism is sources of energy loss and engine troubles, the jiggle valve mechanism is removed from the present thermostat. And a small hole is formed in the flange of the thermostat. Therefore, pressures applied to the outer side and inner side of the main valve become equal  
25 to each other. The spring constant of the return spring is reduced. As a result, the lift up rate is increased in a low temperature range. Furthermore, the thickness of the seal spool is extremely thin

(thickness of between 25 % and 5 % of the diameter of the actuating rod), so that the pressure of the wax for the lift up of the valve is reduced.

Fig. 4 is a diagram showing the lift with respect to the coolant temperature. A line X is the lift of the valve of the present invention and the line Y is the lift of the conventional valve. The reference W shows a lift of the bypass valve 15. The range of the steep curve line is the solid wax state.

The main valve of the conventional valve Y opens at 72°C, the lift at the end temperature 87°C of the solid wax state is merely 9.6 mm. Thereafter, the lift up rate reduces because of the liquid wax state, and when the lift becomes 12 mm, the coolant temperature reaches such a high temperature as 123°C.

The main valve of the present invention also opens at 72°C, the lift becomes 6 mm by a small temperature increase of 9°C. When the lift reaches 12 mm, the coolant temperature is 85°C as shown by the line X, which is only increase of 4°C. The coolant temperature of 85°C is within the range of the solid wax state.

When the automobile mounting the thermostat of the present invention is driven at 80 Km/h, the coolant temperature increases to 77.5°C. However, when the speed reaches to 150 Km/h, the coolant temperature decreases to 70.5°C, because the radiator is cooled by a strong wind. Therefore, the cooling system is provided such that the cooling fan starts at an upper limit

temperature of 81°C and stops at 75°C.

The line Z'-Z of Fig. 4 shows the upper limit of 81°C. The hatched area shows the difference between the flow rate of the coolant passing the main valve of the present invention and the flow rate in the conventional valve. The lift of the line X at 81°C is 6 mm and the lift of the line Y is 3 mm. Therefore, the flow rate of X is two times as much as the flow rate of Y.

The flow rate at the lift of 6 mm of X which is in the solid wax state corresponds to the flow rate at the lift of 12 mm of Y which is included in the liquid wax state. Thus, the thermostat of the present invention uses only 50 % of the own power at the lift of 12 mm. Therefore, even if an automobile mounting the thermostat of the present invention is driven at 150 Km/h, there remains the power of 50 %.

However, the conventional thermostat of the line Y enters the liquid wax state range after 86°C (lift 9.3 mm), where the lift increasing rate largely decreases. The coolant temperature of 37°C (123°C - 86°C) is consumed in vain in the period from the lift 9.3 to 12 mm. The thermostat of the present invention demonstrates a double power with half of source, remaining power of 50 %.

The coolant flowing through a small hole on the flange needs not worry about taking a longer warm up period for a idling because the cold start fuel injector is provided in the throttle body controlled by a computer compensate a time loss for a idling.

In the cooling system using the thermostat of the present invention, when the coolant temperature exceeds 81°C, the cooling fan operates. Since the flow rate of the coolant at 81°C is the double of that of the  
5 conventional thermostat, the coolant temperature quickly decreases. Therefore, the coolant temperature is kept 81°C.

The upper limit for the cooling fan is not limited to 81°C. It is desirable to set the temperature to an  
10 effective value as low as possible dependent on tests.

The high coolant temperature of the conventional thermostat causes various problems such as an increase of fuel consumption and aggravation of the emission.

#### SUMMARY OF THE INVENTION

15 An object of the present invention is to provide a thermostat for an automotive engine cooling system which may decrease the upper limit temperature of the coolant.

According to the present invention, there is provided a thermostat for an automotive engine cooling  
20 system, the thermostat having a housing having a flange for attaching thereof to a conduit member, an actuating rod secured to the housing at a first end thereof, a guide member slidably mounted on the actuating rod, a resilient seal spool provided around a second end  
25 portion of the actuating rod and hermetically secured to the guide member, a heat sensitive cylinder housing the seal spool and secured to the guide member, wax pellets provided in the heat sensitive cylinder to enclose the

seal spool, a lubricant oil provided in a space between  
the seal spool and the actuating rod, a main valve  
provided on the guide member, and a return spring for  
urging the main valve to a valve seat formed on the  
5 flange.

The flange has a hole so as to pass a coolant, and  
the thickness of the resilient seal spool is set between  
25 % and 5 % of the diameter of the actuating rod.

The spring constant of the return spring is a half  
10 of the conventional one.

The hole of the flange has a diameter so as to  
equalize the pressure of the coolant outside the main  
valve to the pressure of the coolant inside the main  
valve.

15 These and other objects and features of the present  
invention will become more apparent from the following  
detailed description with reference to the accompanying  
drawings.

#### BRIEF DESCRIPTION OF DRAWINGS

20 Figs. 1 and 2 are sectional side views of a  
thermostat of the present invention;

Fig. 3 is a side view of the thermostat;

Fig. 4 shows a record showing changes of a lift of  
a valve in the thermostat of the present invention and a  
25 lift of a conventional valve with respect to  
temperature;

Fig. 5 shows a sectional view showing a test  
machine for a valve lift;

Fig. 6 is a schematic diagram of a conventional cooling system for an automotive engine;

Fig. 7 shows a thermostat used in the system of Fig. 6; and

5 Fig. 8 shows a record showing changes of temperature and flow rate of the coolant of the conventional system with respect to time.

#### DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT

10 Fig. 1 shows a main valve closing state, and Fig. 2 shows a main valve opening state. A thermostat 1a of the present invention has a housing 10 forming a valve seat 9, a frame 11 secured to a flange 16 of the housing 10, and a thermo-actuator 2.

15 The thermo-actuator 2 comprises an actuating steel rod 3, a guide member 4 slidably mounted on the rod 3, and a resilient seal spool 5 which is hermetically secured to the guide member 4 and slidably engaged with the rod 3. The thickness of a bag portion of the seal spool 5 is between 25 % and 5 % of the diameter of the rod 3. A lubricating oil 6 fills the space between the 20 seal spool 5 and the rod 3.

25 The thickness of the seal spool is determined so that the pressure applied to the inner side of the seal spool becomes equal to the pressure on the outer side, namely lubricant oil pressure becomes equal to the wax pressure.

The seal spool 5 is inserted in a heat sensitive cylinder 8 filled with wax pellets 7. An end of the

cylinder 8 is securely engaged with the guide member 4, thereby forming the thermo-actuator 2.

The rod 3 of the thermo-actuator is secured to the housing 10 at a top portion 14, and the main valve 12 is secured to the guide member 4. A return coil spring 13 disposed surrounding the cylinder 8 is provided between the main valve 12 and the bottom of the frame 11. A bypass valve 15 is slidably mounted on a shaft 14A secured to the cylinder 8 and resiliently held on the shaft 14A by a spiral spring 14a. The flange 16 has a hole 19a (Fig. 3) for communicating the space in the thermostat housing (27 of Fig. 6) with the passage in the thermostat cap (26 of Fig. 6). The diameter of the hole 19a is determined to a value so that the pressure of the coolant in the outer side passage of the main valve 12 (second passage 30 of Fig. 6) becomes equal to the pressure to the coolant in the inner side passage, namely in the housing 10.

As a result, the spring constant of the return spring 13 can be reduced to a half of that of the conventional spring.

Fig. 1 shows the main valve in the close state. When the temperature of the coolant rises in excess of a predetermined value of the thermostat, the wax 7 in the heat conductive cylinder 8 expands. This forces the seal spool 5 against the rod 3. Since the rod 3 is secured to the housing 10, the cylinder 8 is downwardly moved against the force of the spring 13, thereby

opening the main valve 12, and closing the bypass port 32 with the bypass valve 15 (Fig. 2).

When the coolant temperature decreases, the wax contracts. Thus, the coil spring 13 causes the main valve 12 to move to the closed position.

The operating range of the thermostat with respect to the temperature of the wax 7 is divided into a large lift up range of the main valve and a small lift up range. In the large lift up range, the wax is in the solid state where the volume of the wax largely changes. In the small lift up range, the wax is in the liquid state where the volume of the liquid wax changes at a small rate.

In order to increase the thermal efficiency of the automotive engine, it is necessary to increase the valve lift up rate with respect to the coolant temperature, thereby reducing an upper limit of the coolant temperature.

In the thermostat of the present invention, the thickness of the seal spool is extremely thin (thickness of between 25 % and 5 % of the diameter of the rod 3), so that the pressure of the wax for the lift up of the valve is reduced. Furthermore, a small hole (19a) is formed in the flange of the thermostat, so that pressures applied to the outer side and inner side of the main valve become equal to each other. Therefore, the spring constant of the return spring is reduced. As a result, the lift up rate is increased in a low

temperature range.

As set forth above, the main valve of the present invention opens at 72°C. When the lift reaches 12 mm, the coolant temperature is 85°C as shown by the line X  
5 in Fig. 4.

The flow rate at the lift of 6 mm of X which is in the solid wax state corresponds to the flow rate at the lift of 12 mm of Y which is included in the liquid wax state. Thus, the thermostat of the present invention  
10 uses only 50 % of the own power at the lift of 12 mm. Therefore, even if an automobile mounting the thermostat of the present invention is driven at 150 Km/h, there remains the power of 50 %.

Since the flow rate of the coolant at 81°C is the  
15 double of that of the conventional thermostat, the coolant temperature quickly decreases. Therefore, the coolant temperature is kept 81°C.

Fig. 5 shows a test machine for measuring the lift of the main valve with respect to the pressure applied  
20 to the seal spool. In the machine, oil pressure is used instead of wax pressure.

A thermo-actuator 36 is attached in the test machine, cutting the heat sensitive cylinder so as to observe a resilient seal spool 39. The space between  
25 the seal spool 39 and a rod 40, is fitted with a lubricating oil 40. The seal spool 39 can be observed through openings 37 and a transparent plastic pipe 38. The test machine has a slidable rod 44 downwardly urged

by a spring 42. The top of the rod 44 contacts with a rod 43 of a dial indicator (not shown). The oil is supplied from an inlet 35.

Table 1 shows actually measured values for the relation between oil pressure and the lift of the main valve by the test machine of Fig. 5.

Table 1

PRESSURE-LIFT			
PRESSURE	THERMO-ACTUATOR (A) LIFT	THERMO-ACTUATOR (B) LIFT	THERMO-ACTUATOR (C) LIFT
Kg/cm <sup>2</sup>	mm	mm	mm
0	0	0	0
10	0	0	0
20	0	0	0
30	0	0	0
40	0	0	0
50	0	0.4	0.4
60	0	1.5	2.6
70	0	2.8	5.0
80	0.6	6.2	7.8
90	1.6	9.5	10.0
100	2.5	10.0	
110	5.5		
120	8.0		
130	9.5		
140	10.0		
SPRING CONSTANT OF RETURN SPRING: 0.55 Kg/mm			

In the Table 1, a thermo-actuator (A) has rod 3 of 3.8 mm diameter and seal spool 5 of 1.7 mm thickness (45

% of the diameter), a thermo-actuator (B) has the rod of 4.5 mm diameter and the seal spool of 1.25 (25 %), and a thermo-actuator (C) has the rod of 4.5 mm diameter, and the seal spool of 0.225 mm (5 %). The spring constant  
5 of the return spring 13 is 0.55 Kg/mm.

If the thickness of the seal spool 5 is extremely thin as the thermo-actuator (C), the pressure of the lubricant oil 6 in the seal spool becomes equal to the pressure of the wax. Since the resilient seal spool is  
10 in a floating state held by inner and outer equal pressures, the frictional resistance between the seal spool and the rod becomes zero. The rod 3 is relatively lifted up by the pressure of lubricant oil 6 applied to the lower end face of the rod.

15 Since the thermo-actuator (A) has a large thickness of 1.7 mm, the lift is 0.6 mm at the starting pressure of 80 Kg/cm<sup>2</sup>. In order to lift the rod 10 mm against the spring load 15.1 Kg, the pressure of 140 Kg/cm<sup>2</sup> is necessary, which is out of the question.

20 Although the starting pressure for the rod 3 is 50 Kg/cm<sup>2</sup> for both the thermo-actuators (B) and (C) and the lift is the same 0.4 mm, the rod of (C) is lifted up 10 mm by the pressure of 90 Kg/cm<sup>2</sup> because of extremely small thickness 0.225 mm. But for (B), the large  
25 pressure of 100 Kg/cm<sup>2</sup> is necessary.

If the thickness of the seal spool 5 exceeds the thickness of (B), the starting pressure becomes larger than 50 Kg/cm<sup>2</sup>. Therefore, the upper limit of the

thickness is 25 % of the diameter of the rod 3.

The thickness of the seal spool of the thermo-actuator (C) is sufficient. If the thickness becomes smaller, it is difficult to manufacture such a thin seal spool, and the manufacturing cost increases. Therefore, the lower limit of the thickness is 5 % of the diameter of the rod 3.

Table 2 shows actually measured values.

10

Table 2

15

20

PRESSURE-LIFT	
PRESSURE	THERMO-ACTUATOR (D) LIFT
Kg/cm <sup>2</sup>	mm
0	
10	
20	
30	0.3
40	4.0
50	9.5
60	13.5
SPRING CONSTANT OF RETURN SPRING: 0.27 Kg/mm	

25

A thermo-actuator (D) has the same rod diameter and seal spool thickness as the thermo-actuator (C), but the spring constant is reduced from the conventional value of 0.55 Kg/mm to 0.27 Kg/mm which is a half of the conventional value.

The rod is lifted up 0.3 mm at the starting pressure 30 Kg/cm<sup>2</sup>, and 13.5 mm at the pressure 60 Kg/cm<sup>2</sup>. By reducing the thickness of the resilient seal spool to an extremely small value and reducing the  
5 spring constant to a half of conventional value, liquefaction of the wax is promoted, so that the amount of the liquefied wax rapidly increases, thereby rapidly lifting up the main valve by the preeminent synergistic effect of the small thickness of the seal spool and the  
10 small spring constant.

Since the thermostat of the present invention operates at 50 percent of own full power, it operates calmly and quickly, so that the engine operates at small vibration and the life of the engine is extended.

15 Table 3 shows results of endurance tests of four thermostats of the present invention, and Table 4 shows results of endurance test of four conventional thermostats.

20

Table 3

No.	CYCLE	VALVE OPEN TEMP.	LIFT mm	DIFFERENCE FROM INITIAL STAGE	
	340 SEC.	°C	AT 98°C	VALVE OPEN TEMP.	LIFT
5	0	87.2	10.74		
	10,000	86.8	10.78	-0.4	+0.04
	20,000	86.3	10.77	-0.9	+0.03
	30,000	85.8	10.76	-1.4	+0.02
	40,000	86.9	10.77	-0.3	+0.03
10	0	86.5	11.01		
	10,000	86.0	11.02	-0.5	+0.01
	20,000	85.8	11.00	-0.7	-0.01
	30,000	86.0	11.02	-0.5	+0.01
	40,000	86.6	11.00	+0.1	-0.01
15	0	87.4	10.57		
	10,000	87.2	10.63	-0.2	+0.06
	20,000	87.2	10.70	-0.2	+0.13
	30,000	86.6	10.66	-0.8	+0.09
	40,000	86.5	10.64	-0.9	+0.07
20	0	86.6	11.09		
	10,000	85.6	11.12	-1.0	+0.03
	20,000	85.6	11.16	-1.0	+0.07
	30,000	85.8	11.14	-0.8	+0.05
	40,000	86.4	11.16	-0.2	+0.07

Table 4

No.	CYCLE	VALVE OPEN TEMP.	LIFT mm	DIFFERENCE FROM INITIAL STAGE	
	340 SEC.	°C	AT 98°C	VALVE OPEN TEMP.	LIFT
1	0	80.6	8.05		
	10,000	81.0	8.27	+0.4	+0.22
	20,000	80.9	8.26	+0.3	+0.21
	30,000	80.3	8.29	-0.3	+0.24
	40,000	79.8	8.46	-0.8	+0.41
2	0	81.0	8.13		
	10,000	82.0	8.13	+1.0	0
	20,000	80.6	8.45	-0.4	+0.32
	30,000	80.4	8.16	-0.6	+0.03
	40,000	80.6	8.45	-0.4	+0.32
3	0	82.7	7.75		
	10,000	82.5	7.85	-0.2	+0.10
	20,000	82.1	7.78	-0.6	+0.03
	30,000	82.7	7.45	0	-0.30
	40,000	81.5	7.80	-1.2	+0.05
4	0	76.8	9.00		
	10,000	78.3	8.93	+1.5	-0.07
	20,000	78.5	8.60	+1.7	-0.40
	30,000	81.5	7.95	+4.7	-1.05
	40,000	82.6	8.07	+5.8	-0.93

Each of the test was performed 40,000 times by alternating a first test and a second. In the first test, a coolant of below 40°C flows 120 sec., and in the second test, coolant of over 98°C flows 220 sec.

The most important factor for the durability of the thermostat is the variation value of the lift. The variation of the present invention is much smaller than the conventional variation over one figure. The

difference between the lift at the initial stage and that of the final stage is almost zero.

By applying the following improvement to the thermostat of the present invention, it is possible to  
5 further increase the effect of the thermostat.

If the thickness of the resilient seal spool is reduced to an extremely small value, the inner capacity of the heat sensitive cylinder increases, so that the wax pressure decreases. Consequently, it is possible to  
10 reduce the thickness of the cylinder, which causes the inner capacity to be further increased. Therefore, the wax pressure can be reduced in the inverse proportion to the square of the diameter of the rod by increasing the diameter. Thus, it is possible to further reduce the  
15 upper limit temperature 81°C of the coolant temperature.

Furthermore, if a thermostat having wax X line (Fig. 4) which is melted at 69°C is used, the upper limit temperature decreases to 78°C.

In accordance with the present invention, by the  
20 synergistic effect dependent on the very small thickness of the seal spool, and the small spring constant of the return spring, the upper limit is reduced to 81°C or lower, so that the fuel consumption of the engine is reduced, the life of the engine is extended, emission of  
25  $\text{No}_x$  and  $\text{Co}_2$  is decreased, thereby contributing to the prevention of the global warming.

What is claimed is

1. A thermostat for an automotive engine cooling system, the thermostat having a housing having a flange for attaching thereof to a conduit member, an actuating  
5 rod secured to the housing at a first end thereof, a guide member slidably mounted on the actuating rod, a resilient seal spool provided around a second end portion of the actuating rod and hermetically secured to the guide member, a heat sensitive cylinder housing the  
10 seal spool and secured to the guide member, wax pellets provided in the heat sensitive cylinder to enclose the seal spool, a lubricant oil provided in a space between the seal spool and the actuating rod, a main valve provided on the guide member, and a return spring for  
15 urging the main valve to a valve seat formed on the flange, the improvement comprising:

the flange having at least one hole so as to pass a coolant, resulting the spring constant of return spring can be reduced and thickness of the resilient seal spool  
20 set between 25 % and 5 % of the diameter of the actuating rod.

2. The thermostat according to claim 1 wherein the hole of the flange has a diameter so as to equalize the pressure of the coolant outside the main valve to  
25 the pressure of the coolant inside the main valve.

FIG. 1

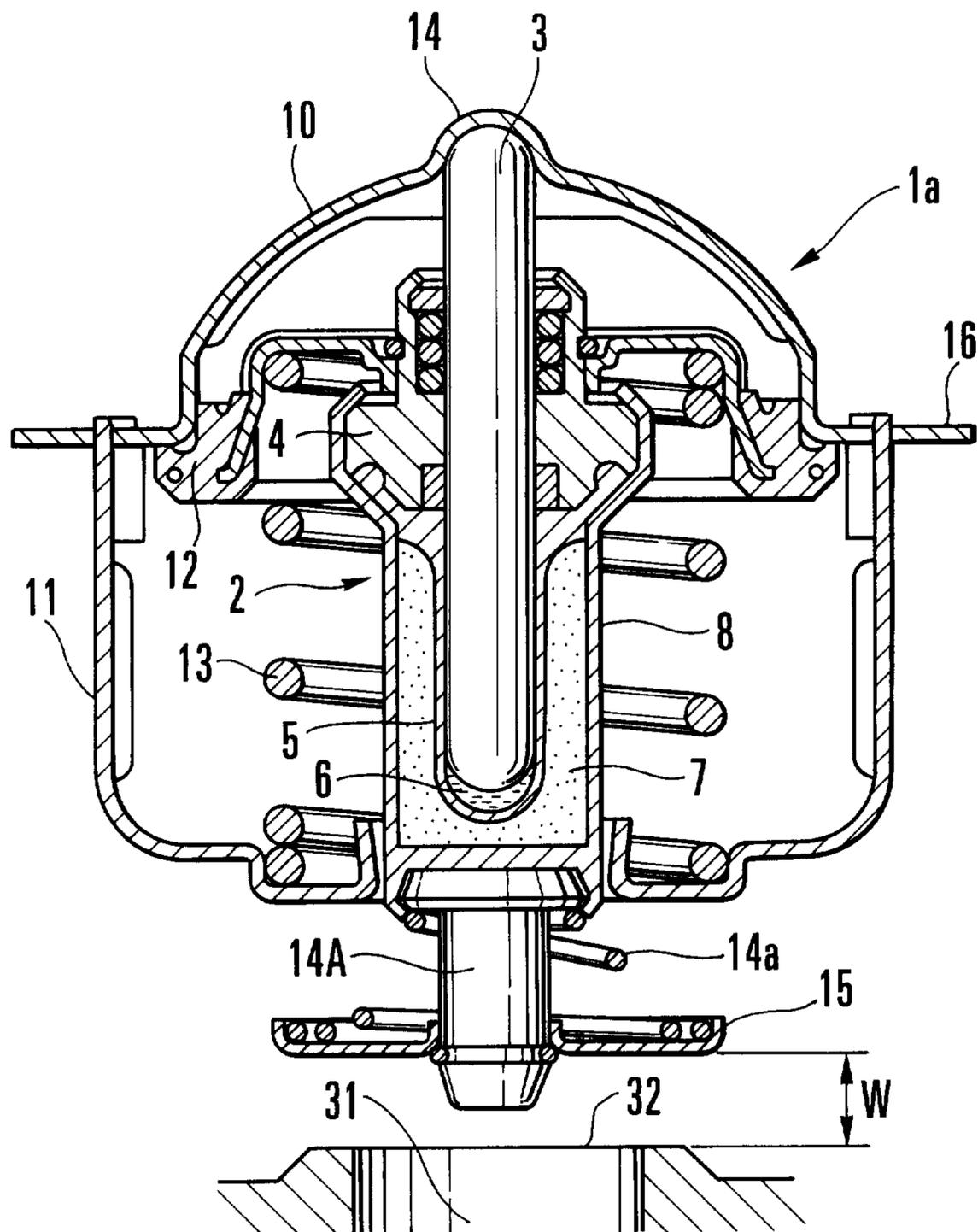


FIG. 2

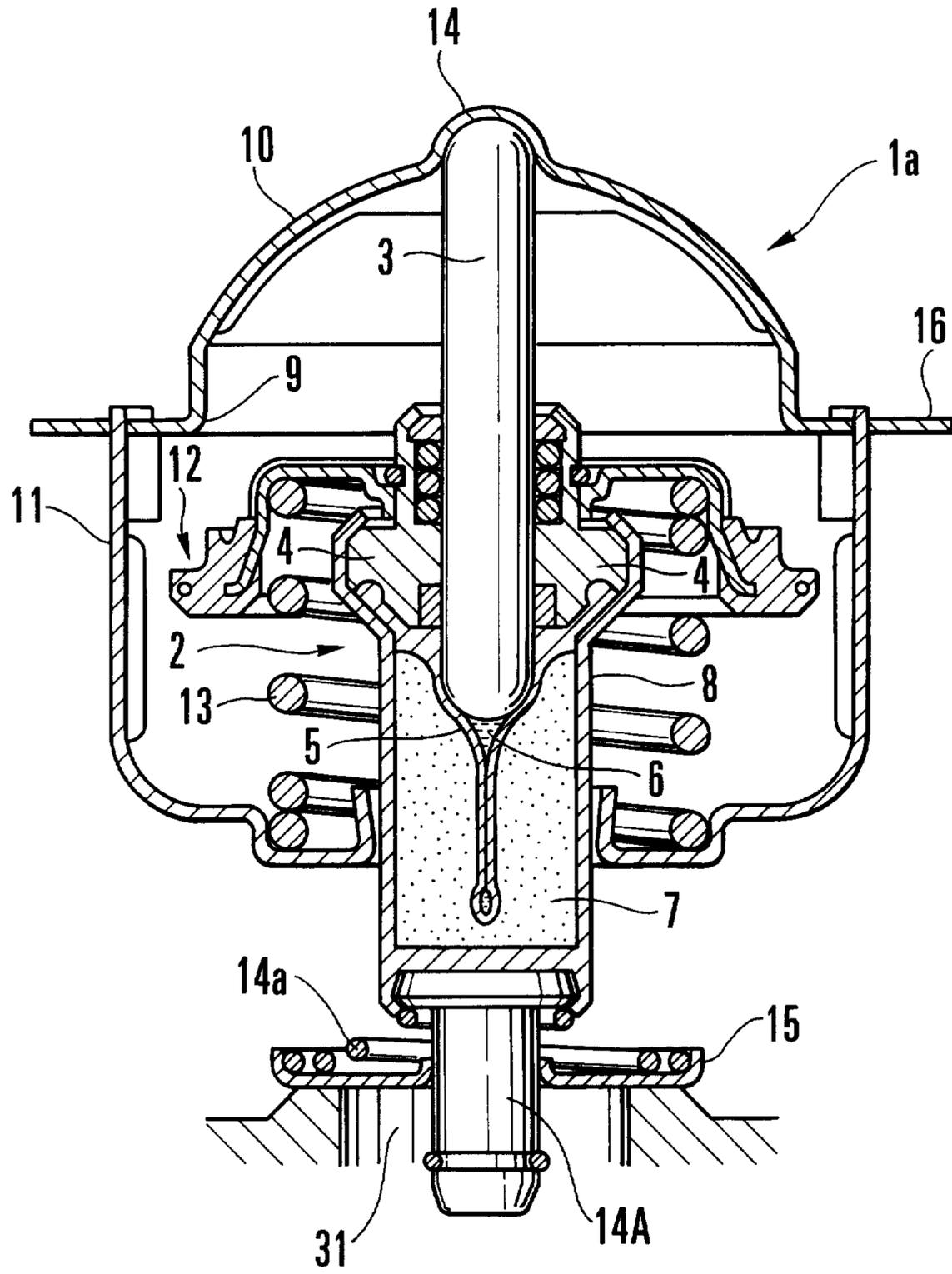


FIG. 3

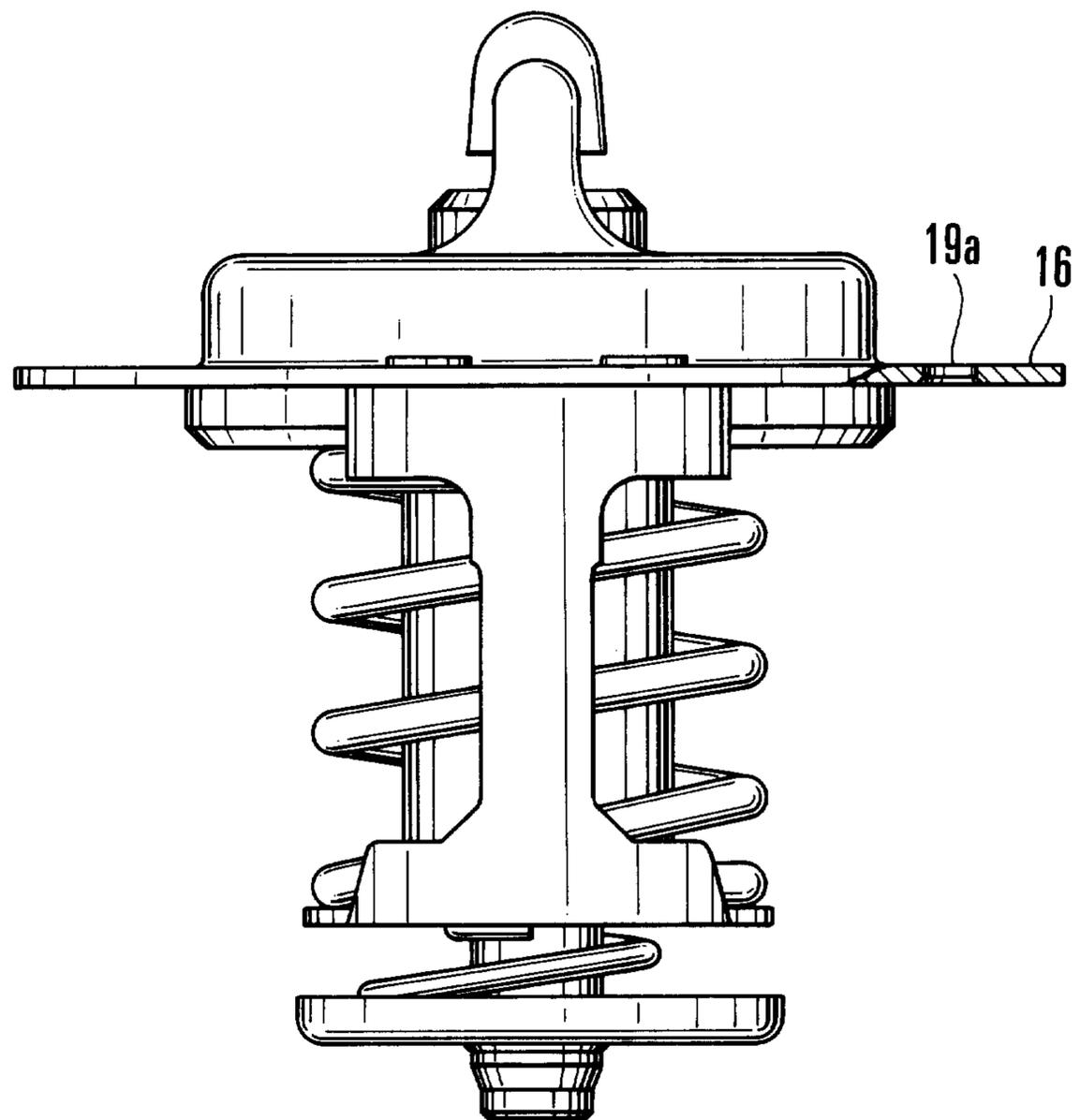


FIG. 4

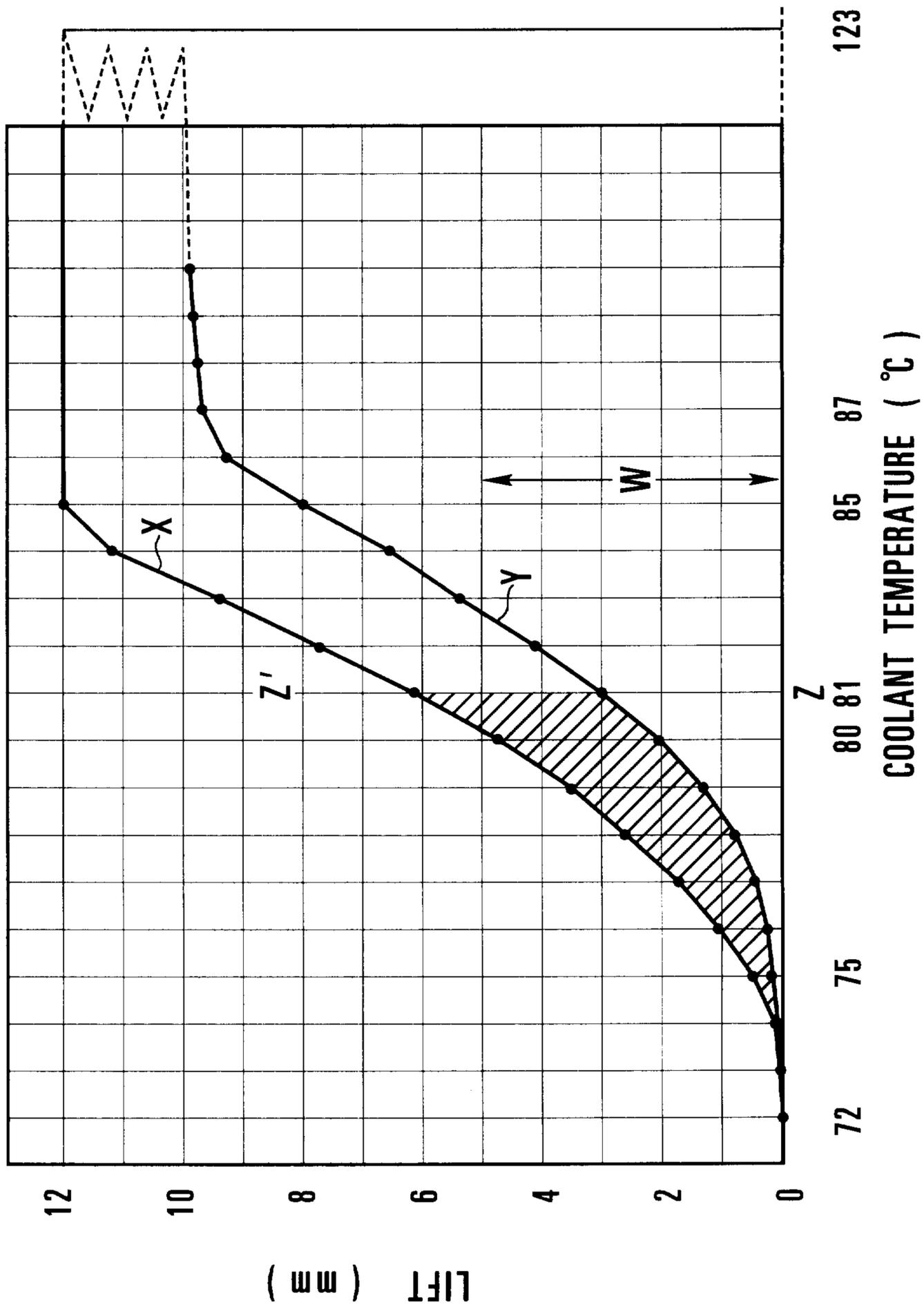


FIG. 5

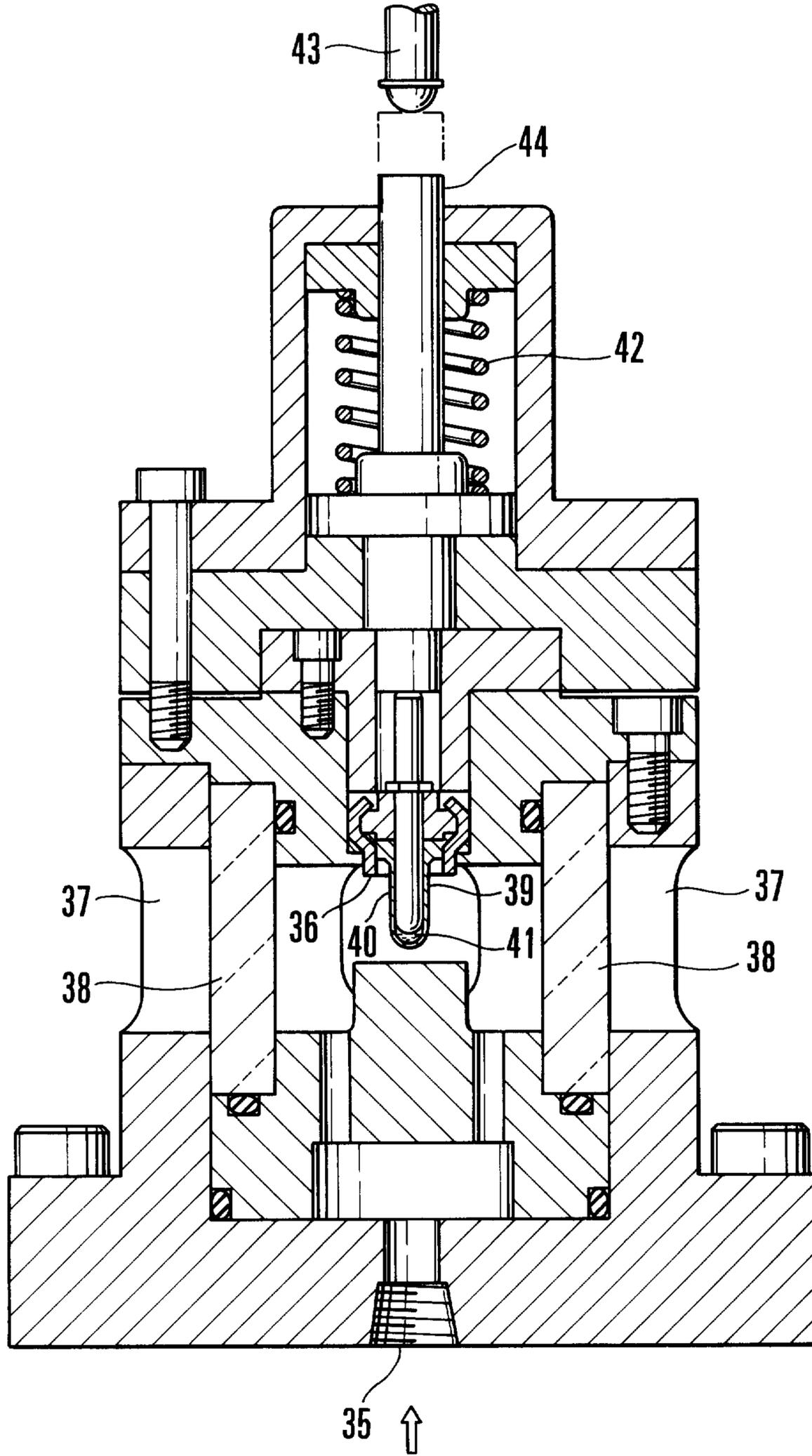




FIG. 7  
PRIOR ART

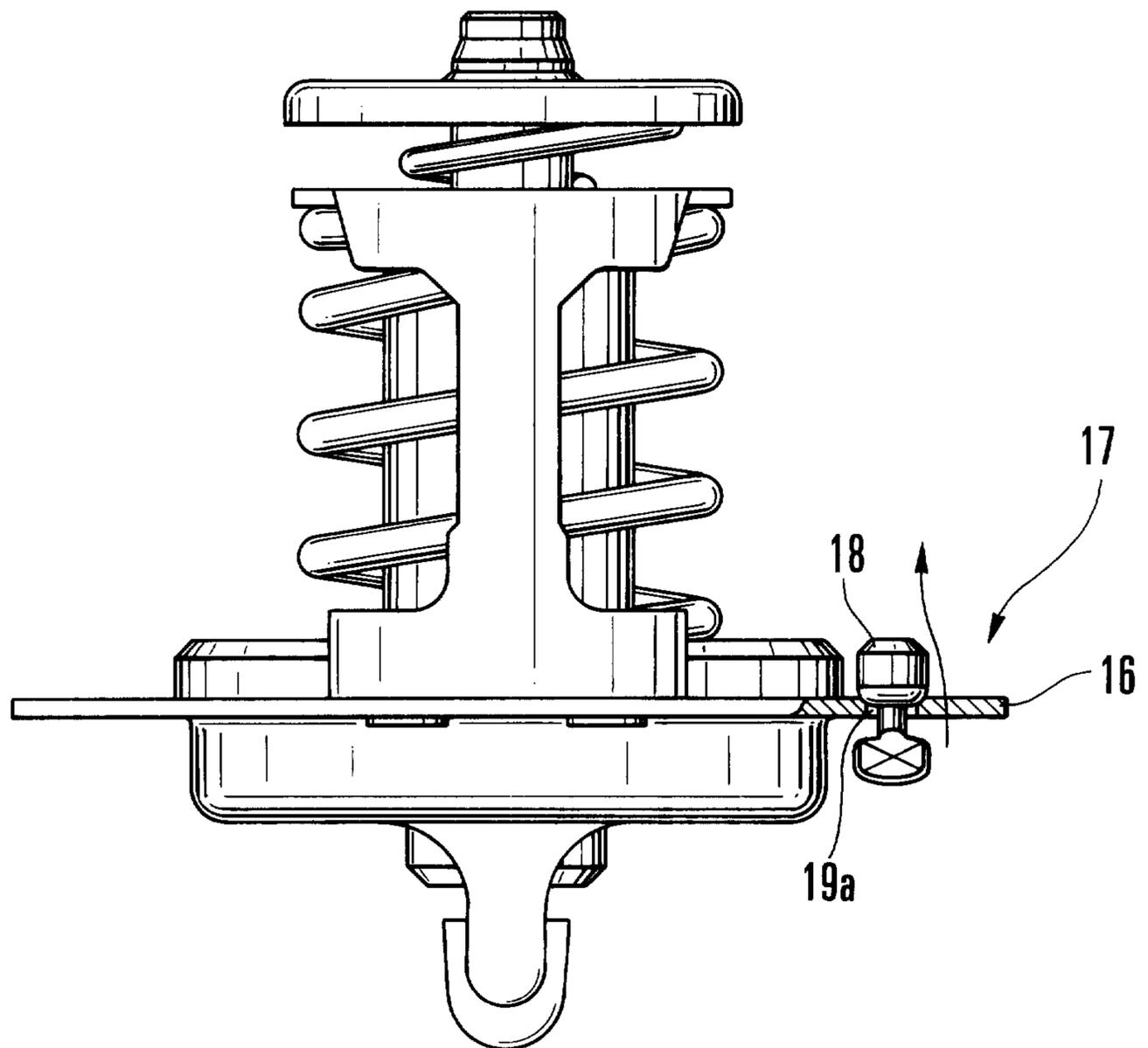


FIG. 8  
PRIOR ART

