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P. J. OLMSTEAD ET AL  
PUMP CONSTRUCTION

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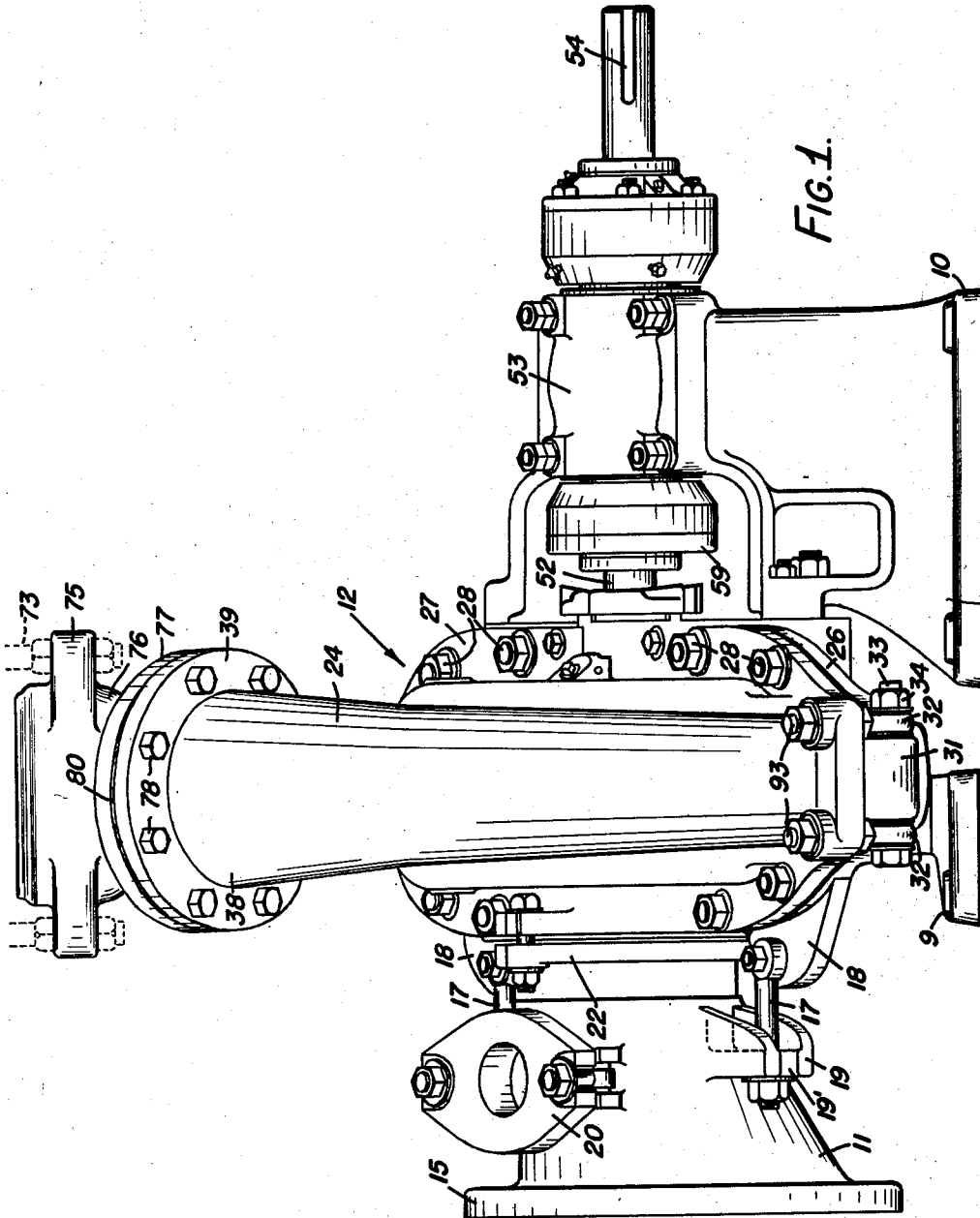


FIG. 1.

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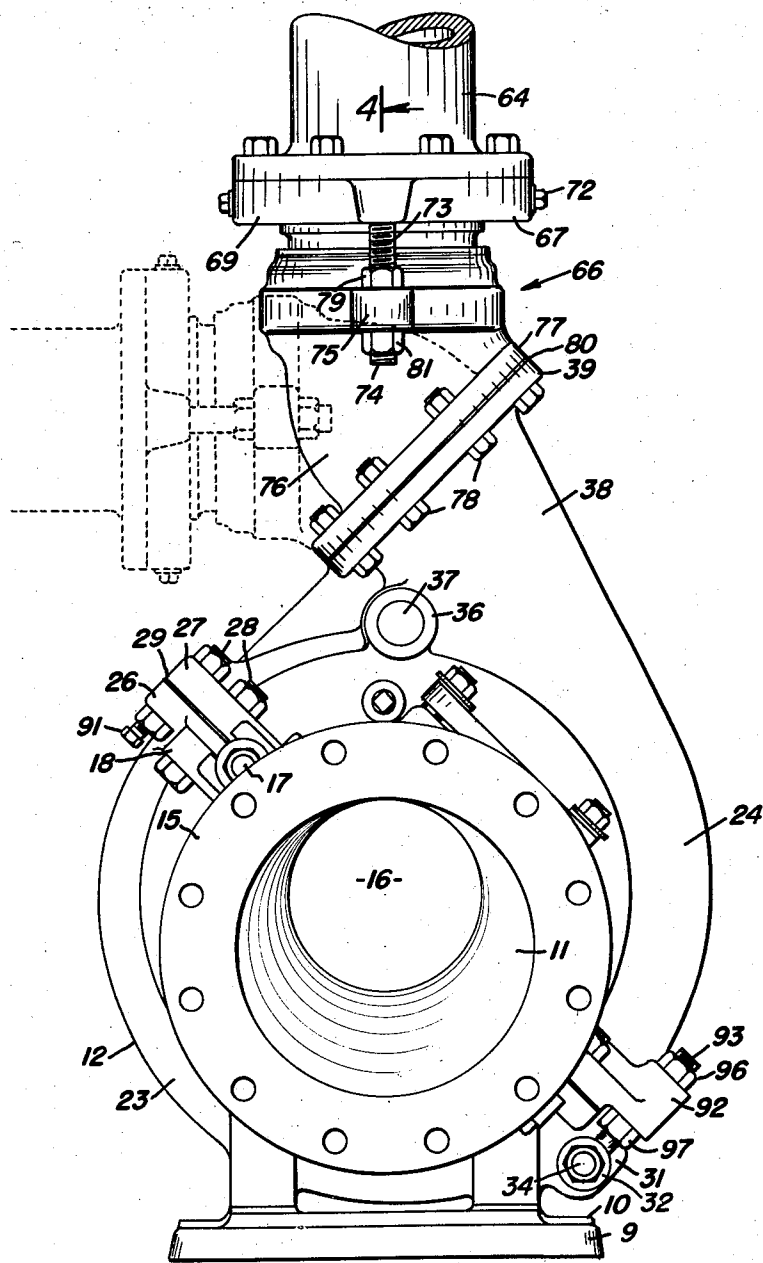


FIG. 2.

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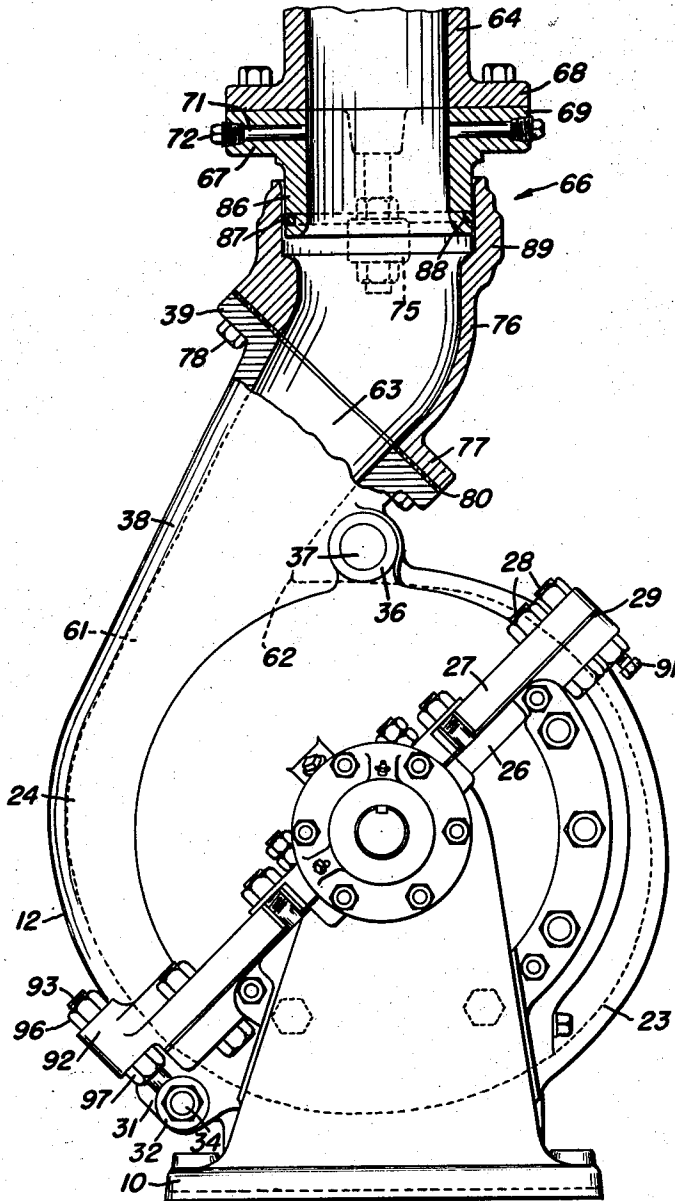


FIG. 3.

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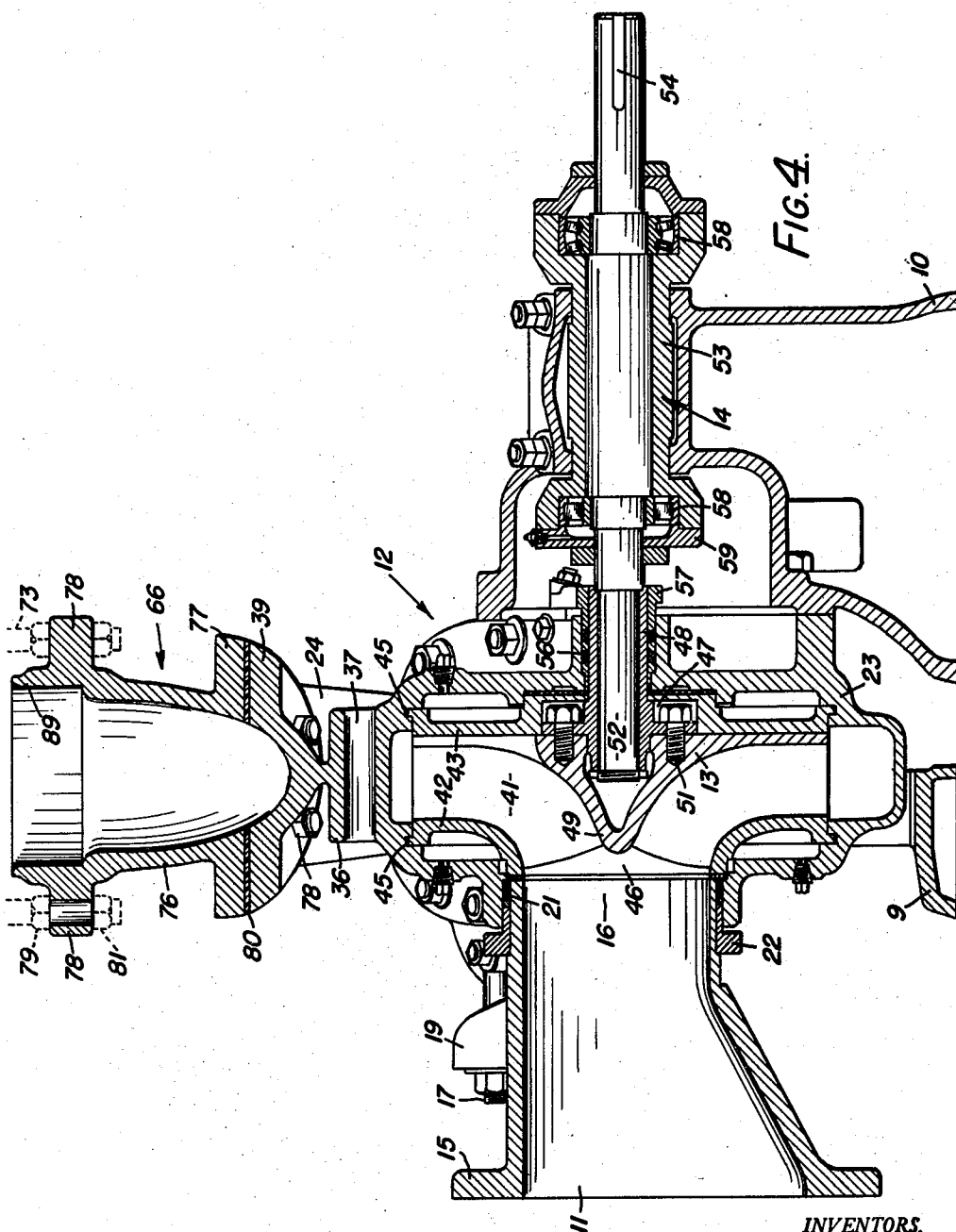
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4 Sheets-Sheet 4



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2,849,960

## PUMP CONSTRUCTION

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Application February 23, 1954, Serial No. 411,681

13 Claims. (Cl. 103—103)

Our invention relates to centrifugal pumps and more particularly to centrifugal stock pumps for use in paper mills.

While the pump of our invention has other applications, particularly where the pump must be self-venting and delays due to the necessity of dismantling the pump for inspection, cleaning or repairs cause large losses in production such as continuous chemical processes, it has been particularly designed for use as a paper stock pump. In a paper mill, all equipment is designed for continuous operation, at least throughout one shift. Continuous operation is dependent upon transporting the stock through a number of successive operations and finally to the paper making machine by means of series of pumps. Any interruption caused by the breakdown of one pump in the series breaks the continuous flows and can tie up the entire production of a plant, or at least the production of one paper machine, resulting in substantial financial losses occasioned by lost production and the idling of employees.

While it is the frequent practice to employ standby pumps to take care of emergencies, duplication of equipment is expensive and requires plant space usually at a premium in a paper mill. The pump of our invention has been particularly designed to enable quick and easy access to the interior of the pump casing for the purpose of freeing a clogged pump, inspection and repair, to avoid the necessity of substantially completely dismantling the pump as is necessary with some types of pumping equipment.

An object of our invention is to provide a pump which is of simple, inexpensive construction and which is designed to enable quick and easy access to the interior of the casing for the purpose of inspection and repair.

Another object of our invention is to provide a centrifugal pump particularly suitable for use as a stock pump in a paper mill in which the pump is self-venting and is split into casing parts along a plane extending at an angle to the vertical and in which the casing parts, upon removing the bolts for holding the casing parts together, may be pivoted with respect to each other without disturbing the suction or discharge piping connected to the pump whereby a minimum of time and effort is required to enable access to the pump casing for the purpose of inspection and repair and subsequent reassembly of the pump to reconnect it into the paper stock flow line.

A further object of our invention is to provide an adapter for use between the discharge of the pump and the discharge piping of the pump wherein the adapter is in sealed floating telescopic relation with the end of the discharge pipe and the pump casing is made in two halves which may be unbolted and shifted with respect to each to expose the interior of the casing, the arrangement of the parts and the construction of the adapter being such that the interior of the pump casing may be quickly exposed for inspection and repair and

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restored to the production line without disturbing the suction or discharge piping.

More particularly, our invention contemplates a centrifugal stock pump which is split along a plane extending at an angle to the vertical and in which the pump discharge is located adjacent the top of the pump for purposes of venting, the pump including an adapter connected to the pump discharge along a plane substantially at right angles to the plane at which the pump casing is split, the construction of the adapter enabling the pump to be readily connected to horizontally or vertically extending discharge piping.

Other objects and advantages of our invention will be particularly set forth in the claims and will be apparent from the following description, when taken in connection with the accompanying drawings, in which:

Fig. 1 is an elevation of the centrifugal pump of our invention showing the suction inlet connection, the pump casing, and the drive shaft for the impeller, together with its supporting bearings;

Fig. 2 is an end elevation looking from the left of Fig. 1;

Fig. 3 is an end elevation looking from the right of Fig. 1; and

Fig. 4 is a sectional view taken substantially on the line 4—4 of Fig. 2 in the direction indicated by the arrows but omitting the fixed part of the floating adapter and omitting the discharge pipe.

The pump of our invention comprises a suction inlet connection 11, a pump casing generally indicated by the numeral 12, a pump impeller 13, an impeller drive shaft assembly generally indicated by the numeral 14, and pedestal supports 9 and 10.

The suction inlet connection 11 (Fig. 4) has a flange 15 for the reception of a corresponding flange on suitable suction piping (not shown). The suction inlet 11 has the lower wall thereof sloping upwardly toward the eye 16 of the impeller to avoid formation of an air pocket adjacent the pump suction. The suction inlet 11 is held in position by swing bolts 17 (Fig. 1) pivoted on bosses 18 rigid with the lower fixed part of the casing 12. The outer ends of the swing bolts extend through slotted lugs 19 integral with the inlet connection 11. The inlet connection upon removing the bolts between the flange 15 and the suction piping and after passing the swing bolts through the slots 19' may be lifted upward when the upper half of the casing 12 is shifted in a manner which will presently appear. However, when access to the pump casing is desired it is not necessary to disconnect the suction pipe nor is it necessary to disturb the swing bolts because the swing bolts are connected to the lower half of the pump casing. The suction inlet connection 11 is provided with a removable handhole cover 20 which constitutes no part of our present invention.

The suction inlet connection 11 is sealed from the adjacent parts of the pump casing 12 by means of packing 21 held in position by suitable annular packing glands 22 bolted to the casing 12 as shown. Although this packing gland is a complete circle, when the two parts of the casing are to be shifted with respect to each other in a manner which will presently appear, it is not necessary to disturb the packing gland or the nuts holding it in position.

Referring now to Figs. 2 and 3, the pump casing 12 comprises a base casing part 23 and a cover casing part 24. These casing parts have flanges 26 and 27 secured together by means of a plurality of through bolts 28 (Fig. 1) which encircle the flanges. A gasket 29 is interposed between the meeting faces of the flanges 26 and 27, and upon drawing up the bolts 28, the casing parts are sealed with respect to each other.

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As shown most clearly in Figs. 2 and 3, the plane of separation of the two halves 23 and 24 of the casing, defined by the faces of the flanges 26 and 27, is at an angle to the horizontal which at least closely approaches the best angle which is 45°. The purpose of this angular position of the plane of separation will presently appear.

As shown most clearly in Fig. 1, the base casing part 23 is provided with a lug 31 which extends between a pair of swing eye bolts 32. A pivot pin 34 extends through aligned bores formed in the lug 31 and the swing eye bolts 32 and is threaded at its ends to receive nuts 33. The pin 34 has a considerable amount of looseness in the bore of the lug 31, the purpose of which will later appear.

The cover casing part 24 has an enlarged boss 36 which has a through bore 37. The bore 37 is adapted to receive a bar or sling to enable the cover casing part to be swung with respect to the base casing part on the pivot pin 34 for purposes which will later appear. The cover casing part extends upwardly as shown at 38 and at its upper end has a flange 39 which extends at an angle to the horizontal. This flange is preferably at an angle which at least approximates the best angle, 45° with respect to the vertical. Thus the flange 39 makes an angle of approximately 90° with respect to the plane of separation of the cover and base casing parts 23 and 24.

Referring now to Fig. 4, the impeller 13 comprises a plurality of vanes or impeller blades 41. In the particular illustration shown, the impeller is of the open type and for the non-clogging purposes required in a paper stock pump, only two vanes are employed. However, if desired, a closed type impeller may be mounted in the casing.

The impeller rotates between wearing plates 42 and 43 which are of generally annular shape and include flanges 45 by which they are recessed in annular grooves formed as a continuous ring in the two matching halves of the casing. The wearing plate 42 has a central annular opening 46 substantially coincident with the area known in the art as the eye 16 of the impeller. The interior walls of the wearing plate 42 are suitably contoured to provide smooth access of the liquid being pumped to the impeller. The wearing plate 43 has a central bore 47 for the reception of a sleeve 48. The hub 49 of the impeller is bolted, as shown at 51, to an enlarged part of the sleeve 48.

The sleeve 48 is fixed with respect to the impeller drive shaft 52 by a key (not shown). The shaft extends through a bearing support 53 carried by the pedestal 10. The projecting end of the shaft is keyed, as shown at 54, for the reception of a coupling for connecting the shaft to the motor shaft of an electric motor or other prime mover (not shown).

The drive shaft assembly further includes a stuffing box 56, a stuffing box gland 57, bearings 58 and a bearing cap 59. These elements constitute no part of our present invention. They are generally of substantially standard or at least well known construction and need not be further described.

Referring now to Fig. 3, the interior walls of the casing parts 23 and 24 are configured so as to form a continuous volute passage 61 indicated in dotted lines extending around the impeller. This volute passage increases in cross-sectional area from the cutwater 62 around the impeller in a clockwise direction, as viewed in Fig. 3, and back to the cutwater 62. At the cutwater the volute merges into a discharge passage 63, the end of which is defined by the flange 39. The pump is constructed so that the cutwater 62 is located adjacent the top of the pump casing to render the pump self-venting. This is extremely important in a paper stock pump as the stock being pumped usually carries a large quantity of entrained air or gas formed by additives to the stock. The construction shown enables any air being pumped into the stock to rise through the pump and flow to the discharge outlet.

In the drawings (Figs. 2 and 3), we have shown the

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lower end of a vertically extending discharge pipe 64. This discharge pipe extends to the point of disposal of the liquid being pumped which may be the next processing station or machine of a paper mill. Between the discharge pipe 64 and the flange 39 of the pump casing, we employ what may be termed an adapter generally indicated by the numeral 66. This adapter comprises a casting 67 rigidly connected by bolts to the flange 68 at the lower end of the discharge pipe 64. In the claims the expression pump discharge connections will be used to designate generally the discharge pipe 64 and the adapter 66.

The casting 67 has an enlarged flange 69 for the reception of bores 71 which are normally closed by plugs 72. Upon removal of the plugs, priming liquid may be introduced into the pump. These plugged openings are also adapted to receive gauges and for the purposes of enabling venting of the pump. The flange 69 has a pair of studs 73 (Fig. 2) anchored therein, the lower ends of which are threaded as shown at 74.

The adapter 66 includes a second casting 76 (Fig. 4) which has a flange 77 secured by means of bolts 78 to the flange 39 of the pump casing. A gasket 80 is interposed between the flanges 77 and 39 to maintain a seal between these parts. The upper end of the casting 76 (Fig. 6) has a pair of outwardly extending bosses 75 which are bored for the reception of the studs 73. Nuts 79 and 81 are applied to the threads 74 of the studs 73 on the upper and lower sides of the bosses 75.

The lower end of the casting 67 is provided with a cylindrical extension 86 (Fig. 3), the exterior wall of which is machined and has an annular groove 87 for the reception of an O-ring packing 88. The casting 76 has an upwardly extending extension 89 of cylindrical shape with an interior machined surface adapted to fit over the exterior machined surface of the extension 86. Thus the castings 67 and 76 are in floating telescopic relation to each other and the slight annular clearance between their telescopic parts is sealed by the gasket 88. The O-ring packing 88 forms an effective seal and the pressure of the liquid being pumped adds to the effectiveness of this seal. The telescopic relationship of the castings 67 and 76 also provides an effective expansion joint to take care of expansion and contraction resulting from temperature changes.

Referring now to Figs. 2 and 3, when the pump has been in service for a period of time, the gasket 29 so effectively seals the casing parts that upon removal of the bolts holding the casing parts together, the seal cannot be broken without damage to the gasket. Some form of wedging pry must be used between the meeting faces of the flanges. Considerable time and effort are required to break this seal.

For the purpose of separating the casing parts, the lower casing part is provided with one or more jack screws 91 which extend through the flange 26 and bear against the face of the flange 27. For the further purpose of enabling separation of the casing halves, the pump casing is in addition provided with a boss 92 having a bore through which the bolt ends 93 of the swing eye bolts 32 extend. The bolt ends are threaded for the reception of nuts 96 and 97. As shown in Fig. 1 and as previously mentioned, two swing eye bolts 32—93 are provided. Upon loosening the upper nuts 96 and tightening up on the lower nuts 97, pressure is applied on the boss 92 tending to lift it away from the lug 31. Upon application of pressure also to the jack screws 91, the two parts of the casing may be readily separated from each other, usually without damage to the gasket and in a minimum amount of time.

As previously mentioned, one of the important considerations in the design of a paper stock pump is the ability of the paper mill operators or maintenance personnel to gain access to the casing of the pump for the purpose of freeing a clogged pump, inspection of the interior of the pump and effecting repairs if necessary. The entire operation of a paper mill is dependent upon the

reliability of a whole series of pumps which convey the stock from one station or machine in the paper making process to the succeeding station or machine. Interruption at any point in this flow as a result of pump clogging or pump failure may interrupt the entire operation of a plant, resulting in large losses for each hour that the plant is, for practical purposes, shut down.

While developments in pump design during recent years have tremendously increased the reliability of operation of paper stock pumps, at the same time, notwithstanding all the design precautions that have been taken and all the ingenuity exercised, they do become clogged occasionally and parts do wear out and break. The pump of our invention has been designed to enable plant maintenance men to gain access to the interior of the pump casing for the purpose of freeing a clogged pump, inspection of the interior or repair in a minimum of time and equally to facilitate reassembly of the pump after repair has been effected.

In the pump of our invention, when access to the interior of the pump casing must be had, as previously mentioned, the suction pipe (not shown), the inlet connection 11 and the packing gland 22 need not be disturbed as the upper half of the pump casing lifts off the inlet connection and packing gland. In gaining access to the interior of the pump casing, the nuts and bolts 78 which connect the flanges 39 and 77 are first removed. The nuts 79 (Fig. 4) are then loosened and the nuts 81 tightened. This causes the bosses 78, together with the casting 76, to move upwardly with respect to the casting 67, the two parts telescoping with respect to each other. This movement of the casting 76 breaks the seal at the gasket 80 between the flanges 77 and 39 and allows these parts to be separated.

The nuts and bolts 28 are then removed. Thereafter, pressure is applied to the jack screws 91 and the nuts 96 are loosened and the nuts 97 tightened so as to break the seal between the casing parts along the gasket 29 and along the plane between the flanges 26 and 27. A bar may then be inserted in the bore 37 or a sling employed to enable the casing part 24 to be swung about the pivot pin 34. The looseness in the pivot pin 34 is provided to allow some play so that the parts will clear each other as the upper casing part is swung.

It will be particularly noted that the casing parts swing to a wide extent or through a large angle with respect to each other so that full access to the pump casing may be had. Upon removal of the obstruction or effecting repairs, the pump may be quickly reassembled by reversing the procedure above set forth.

One of the important aspects of our invention is the fact that by the use of an adapter having telescopic parts, the seal between the casting 76 and the flange 39 may be broken without disturbing the discharge pipe 64. This pipe at all times remains connected to the casting 67. Dismantling and reassembly of the pump may be accomplished in a comparatively few minutes without disturbing any pipe connections to the pump and in most cases, by one man with only a hand wrench and a sling. Moreover, the inlet connection 11 and the packing gland 22 remain in position.

A further fact of importance is that the pump of our invention is adapted to be connected to a horizontal discharge pipe. An arrangement with horizontal discharge pipe is shown in dotted lines in Fig. 2. This is accomplished by merely rotating the castings 76 and 67 (the entire adapter unit 66) through 180° and then attaching it to the flange 39. Since the angle at which the flange 39 extends is 45°, rotation of the casting 76 through 180° places the extension 89 in a horizontal position, in a position to receive casting 67. The casting 67 is rigidly and permanently connected to the horizontal discharge pipe at all times and dismantling of the pump is accomplished in the manner previously described.

While we have shown and described the preferred form of our invention, it will be apparent that various changes and modifications may be made therein, particularly in the form and relation of parts, without departing from the spirit of our invention.

We claim:

1. A centrifugal pump connected to the inlet of pump discharge piping comprising, in combination, a casing having a suction inlet and having an impeller therein mounted for rotation about a horizontal axis, a base part and a cover part forming the main parts of said casing, said parts having a plane of separation defined by bolted together flanges extending at an acute angle to the vertical, a pivotal connection between the parts of said casing located adjacent the bottom of the base part, a casing discharge outlet having a flange formed adjacent the top of the cover part, an adapter having a flange bolted to the flange of the discharge outlet and means for connecting said adapter with the inlet of the pump discharge piping in a manner such that upon removal of the bolts from both of said sets of flanges said adapter may be shifted longitudinally with respect to the discharge piping to separate the adapter flange from the discharge outlet flange and the cover part may be swung on said pivotal connection to expose the interior of said casing without disturbing the discharge piping.

2. A centrifugal pump in accordance with claim 1 wherein said sets of flanges extend at an angle of approximately 90° with respect to each other.

3. A centrifugal pump connected to discharge piping comprising, in combination, a casing having a suction inlet and having an impeller mounted therein for rotation about a horizontal axis, a base part and a cover part forming the main parts of said casing, said parts having a plane of separation extending at an angle of approximately 45° to the horizontal, said plane of separation being defined by bolted together flanges, a flanged discharge outlet connection in said cover part, a cut water formed in said cover part adjacent the top of the casing, a discharge volute extending from said cut water around the impeller and terminating in said cover part, an adapter comprising telescopic parts connected to said discharge piping, one of said telescopic parts being flanged and bolted to the flange of said discharge outlet connection, said flanges being substantially at right angles to the flanges between the casing parts, said casing parts having a pivotal connection between them whereby when the bolts of both of said sets of flanges are removed and the parts of the adapter are telescoped with respect to each other the cover part may be swung about said base part to expose the interior of the casing.

4. A centrifugal pump in accordance with claim 3 in which one of the telescopic parts of the adapter is an approximately 45° elbow.

5. A centrifugal pump having pump discharge connections comprising, in combination, a casing having an impeller mounted therein for rotation about a horizontal axis, a base part and a cover part forming the main parts of the casing, said parts having a plane of separation extending at an acute angle to the vertical, an intake pipe connection connected to said casing, a discharge outlet formed in the cover part and connected to said discharge connections and being adjacent the vertical center line of the pump and a pivot between the base part and the cover part adjacent the bottom of the base part, said discharge outlet and said discharge connections having a plane of separation extending at an acute angle to the horizontal, the angle of said plane of separation being such that when the discharge outlet and the discharge connections are disconnected along said plane of separation and the casing parts are disconnected along their plane of separation the cover part may be swung on said pivot and said discharge outlet will swing away from said discharge connections without interference therefrom.

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6. A centrifugal pump in accordance with claim 5 in which a cut water is formed interiorly of the cover part in a position adjacent the top of the casing and the casing parts have formed interiorly thereof matching volute sections which form a volute in which said impeller rotates extending from said cut water around the axis of rotation toward said discharge outlet.

7. A centrifugal pump in accordance with claim 5 in which the plane of separation of said casing parts extends at an angle of not less than 45° to the horizontal.

8. A centrifugal pump in accordance with claim 5 wherein a suction inlet connection is connected to said pump casing in communication with said intake pipe connection and means are provided enabling the separation of said casing parts and the swinging of said cover part without disturbing said inlet connection.

9. A centrifugal pump in accordance with claim 5 in which the plane of separation between said discharge outlet and said discharge connections is substantially 45° with respect to the horizontal and in which the plane of separation of the casing parts extends at an angle of not substantially less than 45° to the horizontal.

10. A centrifugal pump adapted to be connected to a discharge pipe comprising, in combination, a casing having a suction inlet connection and having an impeller therein for rotation about a horizontal axis, a base part and a cover part forming the main parts of said casing, said parts having a plane of separation extending at an angle of not substantially less than 45° with respect to the horizontal, said plane being defined by bolted together flanges, said casing parts having a pivot connection between them located adjacent the bottom of said base part, said cover part having a flanged discharge outlet, an adapter having a through passage and having a flange connected to the flange of the discharge outlet with the plane of said flanges extending at an angle of substantially 45° with respect to the horizontal, said adapter having telescopically shiftable parts with one of said parts being connected to the discharge pipe, said telescopically shiftable parts enabling the flange of the adapter to be separated from the flange of the discharge outlet and moved away therefrom so that when the casing parts are separated along their plane of separation the cover part may be swung on said pivot without interference between the flange of the discharge outlet and the flange of the adapter.

11. A pump in accordance with claim 10 in which the passage extending through the adapter turns through 180° with respect to the discharge outlet flange for connection of the adapter to a horizontal discharge pipe.

12. A centrifugal pump for connection to a discharge pipe comprising, in combination, a pump casing having an impeller mounted therein, a discharge passage formed in said casing into which said impeller discharges, a

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discharge outlet terminating in a flange, said discharge passage and said discharge outlet being in communication with each other and said discharge outlet extending in an upward direction, an adapter having a flange for connection to the discharge outlet flange, a gasket interposed between said flanges, said adapter having a through opening and being connected to the discharge pipe, said adapter having telescopically shiftable parts to enable the flange of the adapter to be disconnected from the discharge outlet flange and shifted away therefrom to provide a space between the flanges whereby the gasket between the flanges may be replaced without disturbing the discharge pipe, the plane of the flanges extending at an angle of substantially 45° with respect to each other enabling the adapter to be turned through 180° with respect to the discharge outlet flange and connected to a substantially horizontally extending discharge pipe.

13. A centrifugal pump for connection to a discharge pipe comprising, in combination, a pump casing having an impeller mounted therein, a discharge passage formed in said casing into which said impeller discharges, a discharge outlet terminating in a flange, said discharge passage and said discharge outlet being in communication with each other and said discharge outlet extending in an upward direction, an adapter having a flange for connection to the discharge outlet flange, a gasket interposed between said flanges, said adapter having a through opening and being connected to the discharge pipe, said adapter having telescopically shiftable parts to enable the flange of the adapter to be disconnected from the discharge outlet flange and shifted away therefrom to provide a space between the flanges whereby the gasket between the flanges may be replaced without disturbing the discharge pipe, the said telescopically shiftable parts having sealing means between them and means for exerting a mechanical force on the flanged part of the adapter to "break" the gasket seal to enable separation of the flanges.

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