

# United States Patent [19]

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[54] WELL PACKER VALVE ARRANGEMENT

[56]

### References Cited

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[75] Inventors: **Edward T. Wood, Kingwood; Robert E. Snyder, Houston, both of Tex.**

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3,373,820	3/1968	Robinson et al.	166/187
3,427,651	2/1969	Billstein et al.	166/187 X
3,437,142	4/1969	Conover	277/34 X
3,503,445	3/1970	Cochrum et al.	166/151 X
3,542,127	11/1970	Malone	277/34 X
3,581,816	6/1971	Malone	166/187
3,776,308	12/1973	Malone	166/187
3,818,922	6/1974	Malone	166/187 X
4,063,427	12/1977	Hoffman	166/187 X
4,260,164	4/1981	Baker et al.	277/34

[73] Assignee: **Completion Tool Company, Houston, Tex.**

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Primary Examiner—Robert S. Ward, Jr.

### Related U.S. Patent Documents

Reissue of:

[64] Patent No.: **4,402,517**  
 Issued: **Sep. 6, 1983**  
 Appl. No.: **408,123**  
 Filed: **Aug. 13, 1982**

[51] Int. Cl.<sup>4</sup> ..... **F16J 15/46; E21B 33/12**

[52] U.S. Cl. .... **277/34; 277/31; 166/122; 166/187**

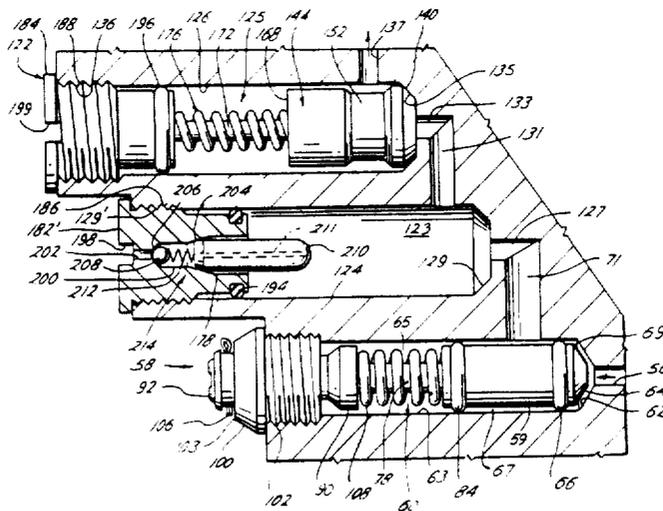
[58] Field of Search ..... **277/3, 30, 31, 34, 34.3, 277/34.6; 166/120, 122, 141, 151, 166, 187, 212, 244 R**

[57]

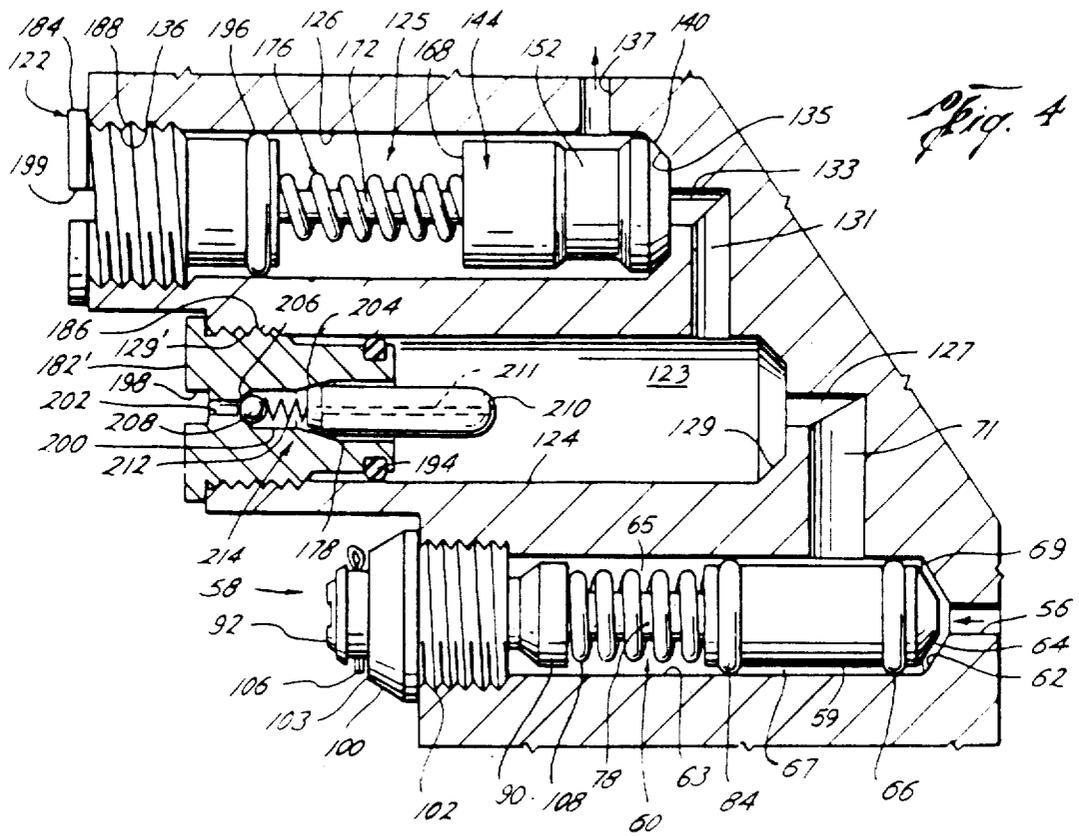
### ABSTRACT

A valve system for use in inflating packers mounted on mandrels. The valve system uses one or more valves to permit, through the use of seals, the flow of fluid from the interior of a tubular mandrel to the interior of the inflatable packer when pressure applied in the mandrel exceeds at least a minimum pressure. The differential pressure across reciprocating seals is minimized through exposure of one or both sides, directly or indirectly, to the external pressure of the mandrel and packer, the exposure including the use of a check valve to permit flow from the exterior of the mandrel.

**7 Claims, 4 Drawing Figures**







## WELL PACKER VALVE ARRANGEMENT

Matter enclosed in heavy brackets [ ] appears in the original patent but forms no part of this reissue specification; matter printed in italics indicates the additions made by reissue.

## BACKGROUND OF THE INVENTION

## 1. Field of the Invention

The present invention relates to packer inflation systems and more particularly to the valves which control the inflation of packers.

## 2. Description of the Prior Art

The control of the inflation of well packers is important to obtain integrity between the packer and the well bore for purposes of working within the bore. It is known in the art to inflate packers by various mechanisms. See, for example, U.S. Pat. No. 3,503,445, issued Mar. 31, 1970, to K. L. Cochran et al., entitled "Well Control During Drilling Operations"; U.S. Pat. No. 3,351,349, issued Nov. 7, 1967, to D. V. Chenoweth, entitled "Hydraulically Expandable Well Packer"; U.S. Pat. No. 3,373,820, issued Mar. 19, 1968, to L. H. Robinson, Jr. et al., entitled "Apparatus for Drilling with a Gaseous Drilling Fluid".

In U.S. Pat. No. 3,437,142, issued Apr. 8, 1969, to George E. Conover, entitled "Inflatable Packers for External Use on Casing and Liners and Method of Use", there is disclosed an inflatable packer for external use on tubular members such as casings, liners, and the like. A valving arrangement is disclosed therein for containing fluid within the interior of the inflatable member after it has been inflated to prevent its return to the tubular member.

Arrangements of valving have been known in the prior art to prevent further communication between the interior of the tubular member and the interior of the inflatable element after the inflatable element has been inflated and set in a well bore. See, for example, U.S. Pat. No. 3,427,651, issued Feb. 11, 1969, to W. J. Bielsstein et al., entitled "Well Control"; U.S. Pat. No. 3,542,127, issued Nov. 24, 1970, to Bill C. Malone, entitled "Reinforced Inflatable Packer with Expandable Back-up Skirts for End Portions"; U.S. Pat. No. 3,581,816, issued June 1, 1971, to Bill C. Malone, entitled "Permanent Set Inflatable Element"; U.S. Pat. No. 3,818,922, issued June 25, 1974, to Billy C. Malone, entitled "Safety Valve Arrangement for Controlling Communication Between the Interior and Exterior of a Tubular Member"; and U.S. Pat. No. 3,776,308, issued Dec. 4, 1973, to Bill C. Malone, entitled "Safety Valve Arrangement for Controlling Communication Between the Interior and Exterior of a Tubular Member".

Inflatable packers have also been used in other operations, such as sealing the annular space between a jacket and a piling. See for example U.S. Pat. No. 4,063,427, issued Dec. 20, 1977, to Erwin E. Hoffman, entitled "Seal Arrangement and Flow Control Means Therefor".

The seals that are used in valves, such as in Malone, are usually hardened rubber. Such rubber tends to extrude under extreme pressure differential across the rubber and cause friction between rubber and metal that adversely affects valve operation. None of the prior art, however, provides for mechanism for equalizing pres-

ures across the seals of the valves used to inflate packers to prevent such extrusion.

## SUMMARY OF THE INVENTION

The present invention utilizes a unique arrangement of sealing mechanisms in conjunction with a valve or valves to permit the inflation of an inflatable packer element while at the same time equalizing pressure around the rubber seals of the valve or valves to prevent distortion of the seals from undue high differential pressure, and the resulting friction.

The present invention, like the prior art, is constructed and arranged so that the valve or valves remain seated to prevent communication between the interior of a tubular member and the interior of an inflatable element carried on the exterior of the tubular member until at least a predetermined pressure has been reached. This reduces the possibility of premature inflation of the inflatable element by sudden pressure changes or pressure surges which may occur within the tubular member as the tubular member is being positioned within a well bore.

However, the valve arrangement of the inflation system of the present invention includes an appropriate check valve arrangement of a portion of the valve structure to compensate for bore pressure to prevent extrusion from undue high differential pressures across the seals of certain rubber seals which must move in the valving operation.

## BRIEF DESCRIPTION OF THE DRAWINGS

For a further understanding of the nature and objects of the present invention, reference should be had to the following detailed description, taken in conjunction with the accompanying drawings, in which like parts are given like reference numerals and wherein:

FIG. 1 is a cross-section of a packer showing the three-valve collar for inflation of the packing;

FIG. 2 is an enlarged cross-section of the valve arrangement of FIG. 1 taken along section line 2—2 of FIG. 1;

FIG. 3 is an enlarged cross-sectional view of the three valves of a three-valve arrangement within the three-valve collar of the prior art; and

FIG. 4 is an enlarged cross-sectional view of three valves of a three-valve arrangement of the present invention within the three valve collar.

## DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT

A tubular [member] inflatable packer 10 is shown in FIGS. 1 and 2. This type of [member] packer could be used for any of the embodiments of the present invention and is specifically illustrated for embodiments 1 and 2, [and may be] as a casing packer. [Member] The tubular inflatable packer 10 includes a short casing joint or casing sub 12 for connection to other tubular members and is secured by suitable means, such as threads as illustrated in FIG. 1, to a valve collar 14 secured to [the body] a tubular pipe member or mandrel 11 [of the tubular member 10]. It should be noted that in one aspect of the present invention, the valve collar 14 could also be and is preferably secured to [the] a sub 36 [of] at the other end of [body] the tubular pipe member 11 [shown in FIG. 1]. [Valve] The valve collar 14 includes a valve mechanism 16 or system of valves and passageways (See FIG. 2) for [communicating] placing fluid [from the interior 21 of tubular member 10 to

the] in a bore or interior 21 of the pipe member 11 in fluid communication with a fluid channel or chamber 20 (See FIG. 2) [leading to the] under an inflatable[, or] packing[,] element [22] 30 carried externally on the tubular pipe member [10] 11.

The inflatable packing element [22] 30 includes spaced apart upper and lower annular packer heads 24, 26. [Lower] The lower packer head 26 is secured to the valve collar 14 [Upper] while the upper packer head 24 is secured to a top or upper collar 35. [Inflatable] The inflatable packing element [22] 30 extends between the packer heads 24, 26 and is also secured to [mandrel 28] the pipe member 11 which extends along the inside surface of the packing element [22] 30 between the valve collar 14 [to] and the upper collar 35 [where mandrel 28 is connected by threading or other means]. The inflatable packing element 30 may be of any suitable length and is [provided with] an elastomer cover [30] and [two sets of] with upper and lower sets of steel anti-extrusion ribs 32. [Ribs] The ribs 32 are connected to the elastomer cover [30,] such as, for example, [vulcanized into the rubber, and extend therein.] by vulcanizing the elastomer cover to the ribs 32 so that the ribs 32 extend into the ends of the elastomer cover. Each set of ribs 32 is connected to a steel back-up sleeve 34, and one set of ribs is connected to the valve collar 14 while the other set of ribs is connected to the upper valve collar 35. [Sleeve] The back-up sleeve 34 is also connected to [packing element 22,] the elastomer cover, such as vulcanized with the rubber, and to the valve collar 14. [A] The tubular sub 36 is connected to the [other portion of] upper collar 35 for use with other tubular members in a string of pipe or casing (not shown).

[A] As shown in FIG. 2, a first set of annular grooves 38 is formed [on] in the valve collar 14. The set of grooves 38 includes internal, circumferential annular grooves 40, 42 [formed in valve collar 14. Grooves 40, 42 are partially] spaced longitudinally apart from one another and covered by a juxtaposed screen sleeve 44. [Sleeve] The screen sleeve 44 includes a hole 46 [covered by] which receives a knock-off rod or plug 50, usually constructed of plastic, to isolate the valve system from fluid under pressure in the [interior] bore 21 of the [member 10] pipe member 11 during running of the inflatable packer 10 into a well bore containing fluid.

[Groove 42 terminates in] A port 52 [extending] extends partially through the wall of the valve collar 14 and [connecting to] connects a passageway 54 [Passageway] to the groove 42. The passageway 54 extends along the center of the valve collar 14 [to] between a port 56 to a valve in the valve mechanism 16 and the port [56 of the valve system] 52 (See FIG. 3).

#### —PRIOR ART—

[Shear] In a prior art valve mechanism as shown in FIG. 3, a shear valve 58 [(FIG. 3, FIG. 4)] is in fluid communication with the port 56 [via insertion of], the shear valve 58 being in a valve pocket 60 [pocket 60] formed by a cylindrical bore 63 in the wall of the valve collar. The shear valve 58 includes a movable valve element or valve body 59 slidably disposed in the valve pocket 60. The valve pocket 60 is formed in the valve collar 14 by drilling [of] or other means. [Valve] One end of the valve pocket 60 is in fluid communication with the port 56 [Pocket 60 forms] and this end of the valve pocket 60 has an angled valve seat 62 [at the end of pocket 60 in direct fluid communication with] disposed around the port 56. The [other] open end of the valve pocket 60

[is threaded with] has threads 61. [Pocket 60 is cylindrical in shape having upper surface 63 of one] The movable valve element 59 has a cylindrical first surface 65a with a first diameter [in] forming an upper or first chamber 65 with the surface wall of the valve pocket 60 and [coaxial lower] has a cylindrical second surface 67 [of] with a second, [smaller] diameter [in] forming a lower or second chamber 69 [Upper] with the surface wall of the valve pocket 60. The lower or second chamber [63] 69 has an opening to a lateral passageway 71 [at one end] which extends further into the valve collar 14.

[Valve 58 includes a cylindrical shaped] The movable valve element or valve body 59 [with] defining the cylindrical second surface 67 terminates at an end portion 64 shaped to [fit in] compliment the valve seat 62. A T-seal, or other suitable seal [,] 66 is [included along] disposed on the [circumference 73 of] second surface 67 the valve body 59 in a groove 68 [of] near the end portion 64. [Seal] The seal 66 is adapted to engage the surface wall [67 of] of the valve pocket in the lower or first chamber 69 [substantially parallel to the circumference 73]. [A threaded] The movable valve element or valve body 59 has a bore 70 [having] with internal threads 74 [is formed longitudinally along the lower portion of body 59. End 64 is connected by] to connect to the end portion 64 by virtue of external threads 72, or other suitable means[, to internal threads 74 of the longitudinal bore 70]. The movable valve element or valve body 59, as illustrated, is reduced in size at the end opposite to the end portion 64 to form a valve stem 78 with a first shoulder 80 [formed at the juncture of valve stem 78 and] on the valve body 59. A suitable seal 84, such as an O-ring, is arranged in a groove 86 on [the upper] a portion of the valve body 59 between the end portion 64 and the first shoulder 80. [Seal] The seal 84 is adapted to seal against the [upper] surface [63 of upper chamber 65 of pocket 60] wall and the groove 86 [and separate the upper or first chamber 65 from the lower or second chamber 69].

[Valve] The valve stem 78 terminates at [its top] an end 88 which [is adjacent] abuts an end surface 98 in a sleeve 94 in a collet 90. [Collet] The collet 90 has a thick top section 92 and an elongated tubular sleeve 94 terminating in bell-shaped, lower section 96. [Sections 92 and 94 form an inner end 98 which abuts stem top 88. Collet] The collet 90, which abuts the valve stem 78 at [its inner end] the surface 98 [,] is retained in the valve pocket 60 by an annular retainer housing 100 which annularly surrounds the collet 90 [Annular] and a shear pin 106 which interconnects the retainer housing 100 to the collet 90. The annular retainer housing 100 has a base 101 with threads 102 [formed on the outer circumference thereof. Threads 102] which mate with the threads 61 [which secured] of the valve pocket 60 and secured the retainer housing 100 to the valve pocket 60. [Housing] The retainer housing 100 further has a bore 97 formed through its base 101 to receive the collet 90 and an opening 116 [at its top] through which the top section 92 of the collet 90 extends.

[A] The shear pin 106 extends through a bore 99 in a notch 103 in [the] an end 104 of the retainer housing 100 and a bore 105 in the end 92 of the collet 90 as shown in FIGS. 3 and 4 to retain the valve [58] body 59 in the seated position with its end portion 64 adjacent the valve seat 62 [to] so that the seal 66 will block off [fluid flow through] fluid and pressure from the port 56 [from] and the [interior 18] bore 21 of the tubular pipe member [10] from access to the passageway 71 and

hence prevent fluid or pressure from reaching the fluid channel 20 leading to the interior of the inflatable packing element 30 [via passageway 71] 30.

A spring 108 [surrounds] is disposed on the valve stem 78 with one end of the spring abutting the shoulder 80 and the other end of the spring abutting [the] an end 110 of the collet 90, such spring 108 being forced to a collapsed position as illustrated when the valve body 59 is in the position as shown in [FIGS. 3 and 4] FIG. 3 of the drawings.

The strength of shear pin 106 will determine the minimum amount of fluid [differential] pressure necessary in the port 56 to unseat the valve body 59 and to open the shear valve 58 and to permit fluid flow through the port 56 from the [interior of tubular] bore 21 of pipe member [10] 11 to the interior of [packer] packing element 30.

[Seals] The seals 66, 84 are positioned on the valve body 59 such that when the valve [58] body 59 is in the seated position as shown in [FIGS. 3 and 4] FIG. 3, the seals 66, 84 prevent any fluid flow [from] or pressure in the port 56 [to] from reaching the passageway 71. [They] The seals 66, 84 also prevent the flow of any fluids or pressure from the exterior of the valve collar 14 [in contact with] from reaching the passageway 71. Fluid in the bore hole [which leak through] by the threads 102 and past the collet 90 in the housing 100 and into the upper or first chamber 65 [to flow into passageway 71]. In addition, when the valve [58] body 59 is in the seated position, the shoulder 80 on the valve body 59 is separated from a bottom 110 of the collet 90 by a sufficient distance [such] so that when the valve [58] body 59 is no longer in the seated position [but] then the shoulder 80 is as close to the shoulder 110 as the [springs] spring 108 will allow[,] and the seal 66 is positioned above the passageway 71[,] so that fluid and pressure is admitted to the passageway 71 from the port 56.

In FIG. 3, the check valves 120 and 122 are substantially identical in construction. [Valves] The check valves 120, 122 are located in valve pockets 123, 125 respectively. Each valve pocket 123, 125 is formed by a cylindrical bore and is substantially cylindrical in shape with walls 124, 126 respectively formed by drilling or other suitable means [of opening with] and each valve pocket 123, 125 opens at one end at the exterior outer surface of the valve collar 14. The other end of the valve pocket 123 terminates at a port 127 which places the valve pocket 123 in fluid communication with the [pocket 123 and] passageway 71. [Pocket] The valve pocket 123 [forms] is formed with an angled valve [seats] seat 129 at the end of the valve pocket 123 in direct fluid communications with the port 127. The other end of the valve pocket 123 is threaded with threads 129'. [Passageway] A passageway 131 also is formed in the valve collar 14 and extends laterally further into valve collar 14 from the wall of the valve pocket 123 and is in fluid communication with the valve pocket 123. The other end of the valve pocket 125 terminates at a port 133 which is in fluid communication with the valve pocket 125 and the passageway 131. [Pocket] The valve pocket 125 [forms] is formed with an angled valve [seats] seat 135 at the end of the valve pocket 125 and is in direct fluid [communications] communication with the port 133. The other end of the valve pocket 125 is threaded with threads 136. [Passageway] A passageway 137 is also formed in the valve collar 14 and extends laterally further into the valve collar 14 from the wall 126 of the

valve pocket 125 and is in fluid communication with the valve pocket 125 and the fluid channel 20. (See FIG. 2)

Each of the [poppet] check valves 120, 122 includes an end portion 138, 140 respectively of elastomer material for engaging on seats 129, 135 respectively formed between ports 127, 133 and the walls of the valve pockets 123, 125 respectively. Each check valve 120, 122 has a movable valve body or element 142, 144 respectively. The general shape of each valve body 142, 144 is cylindrical in configuration. The valve body 142, 144 of each valve pocket 123, 125 has an upper portion 146, 148 respectively and a lower, smaller diameter portion 150, 152 respectively with a swage or annular recess 154, 156 respectively separating the upper and lower portions of a valve body. The tops of elastomer [ends] end portions 138, 140 are fitted into grooves 158, 160 respectively formed circumferentially in lower [ends] portions 150, 152 respectively to hold the elastomer [ends] end portions on the lower portions 150, 152 respectively. A bore 162, 164 is formed [through the] in an end 166, 168 respectively of [valves] the valve bodies 142, 144 [facing] and faces away from the seats 129, 135 and extends substantially through the valve bodies 142, 144 respectively. A valve stem 170, 172 is inserted in [the] a counter bore 162, 164 respectively with a spring 174, 176 in its [collapsed] expanded position circumferentially surrounding the valve stems 170, 172 respectively.

Each valve stem 170, 172 is received in a bore 178, 180 respectively in a retainer housing 182, 184 of the check valves 120, 122 respectively. Each retaining housing 182, 184 is externally threaded with threads 186, 188 adapted to mate with threads 129', 136 respectively of the valve pockets 123, 125 respectively. Each retaining housing 182, 184 also includes [a slot] an annular groove 190, 192 sized to receive a sealing means 194, 196, such as an O-ring, to sealingly engage the walls 124, 126 of the valve pockets 123, 125 and [slots] the grooves 190, 192 respectively. Each retaining housing 182, 184 also includes a [groove] slot 198, 199 respectively cut out in [the head] each retaining housing for external access or removal of valve stems and valve bodies from the valve collar 14.

In operation, when the rod or knock-off plug 50 is still in place, (See FIG. 2) any communication of fluid or pressure from the [interior] bore 21 of tubular pipe member [10] 11 to the fluid port 56 [of any of the prior art or the embodiments] is prevented. This prevents pressure variations or pressure surges from acting through port 56 and unseating the valve body 59 of the shear valve 58 which might prematurely inflate the packing element 30.

When it is desired to actuate the device [of any of the embodiments] and communicate fluid to the channel 20 of the packing element [22] 30 carried on the exterior of the casing or tubular member [10] 11, any suitable means (not shown) may be dropped through member [10] 11 so as to break or shear the rod 50 to permit fluid communication with the [groove set] set of grooves 38.

Thereafter, fluid may be communicated through the set of grooves [40, 42,] 38, the port 52, and the passage 54 to the inlet port 56 between the inner and outer walls of the valve collar 14. The fluid pressure of this fluid acts upon the end portion 64 of the valve [58] body 59, and the pressure within the tubular member [10] 11 may be increased so as to produce a pressure differential between the fluid in the bore 21 of the tubular member 11 and the fluid exterior to the valve collar 14 and thereby

shear the pin 106 whereupon the valve body 59 moves to a position where the seal 66 no longer obstructs the flow of fluid to the passageway 71 from the port 56 thereby permitting fluid flow from the port 56 through passageway 71 to the port 127. This longitudinal movement of the valve body 59 causes the valve stem 78 as well as the collet 90 surrounding the end thereof to move outwardly through the opening 116 of the retainer housing 100, compressing spring 108 between the shoulder 80 and the end 110 of the collet 90. The flow of fluid to the port 127 builds up pressure on the end 138 of the valve body 142. When the pressure on the end portion 138 of the valve body 142 overcomes the break out friction of the end portion 138 and the force required to compress the spring 174, the valve body 142 [rises] moves so that the end portion 138 no longer obstructs the flow of fluid from the port 127 through the passageway 131 to the port 133. The flow of fluid to the port 133, in turn, builds up pressure on the end portion 140, when] of the valve body 144. When the pressure on the end portion 140 overcomes the break out friction of the end portion 140 and the force required to compress the spring 176, the valve body 144 [rises] moves so that the end portion 140 no longer obstructs the flow of fluid from the port 133 to the passageway 137 to the channel 20 and [packer] fluid supplied to the packing element 30 inflates the packing element 30.

Those skilled in the art would believe that the shear pin 106 in the shear valve 58 would shear at a given pressure at the port 56 relative to the pressure exterior of the valve collar 14 depending only on the strength of the shear pin 106. However, this is not the case. At the time the tubular pipe member [10] 11 is lowered into the well, the pressure in the passageway 71 is at atmospheric pressure. The same is true of the pressures in the upper [pocket] or first chamber 65 and the pressure at the port 56. However, as the tubular pipe member [10] 11 is lowered into the well, the pressure in the upper [pocket] or first chamber 65 changes to [that of the exterior of] the pressure in the well because there is no seal [through] to prevent well fluid from leaking by the retainer housing 100 as discussed above. In addition, as pressure within the tubular pipe member [10] 11 increases, the pressure at the valve port 56 increases. However, there is no path for the rising or increasing fluid pressure to enter passageway 71 and to raise [it] the pressure in passageway 71 above atmospheric pressure. Accordingly, while the valve body 59 is seated, the seals 66, 84 will tend to extrude toward the passageway 71 because of the high differential pressure between the upper [pocket] or first chamber 65 and the passageway 71, and the high differential pressure between the lower [pocket] or second chamber 69 and the passageway 71. In such circumstance, the [seal rings] seals 66, 84 [are] become locked and the pressure required in the port 56 to overcome the breakout friction to move the valve body 59 [then goes] is much higher. This is because the O-rings usually used in the prior art [of] embodiment shown in FIG. 3 are designed to only hold 4,000 to 5,000 psi of differential pressure. In deep wells, this breakout friction would be very high and normally a discontinuity in breakout [pressure] friction is exhibited at wells having a depth which exhibit downhole pressures of 5,000 to 6,000 psi. In addition, [as discussed above, the diameter] the annulus of the upper [pocket] chamber 65 is larger than the annulus of the lower [pocket] chamber 69. In the prior art, in order to overcome this difference in [diameter] the annulus,

a sleeve is installed in the upper [pocket] chamber 65. Nevertheless the [sleeves] sleeve may not be perfect and the remaining space in the upper [pocket] chamber 65 is elliptical in shape having a major and a minor diameter both larger than the diameter of the lower [pocket] chamber 69. Therefore, the force of the pressure on the seal 84 in the upper [pocket] chamber 65 is greater than the force by an identical pressure acting on the seal 66 from the valve port 56.

Accordingly, as the well [is deeper and] depth increases, the pressure in upper [pocket] chamber 65 increases[,] and the amount of pressure required at the port 56 to open the valve 58 may be far greater than anticipated [by knowledge of] with respect to the shear strength required of shear pin 106 in order to cause shear pin 106 to shear at a selected pressure.

#### EMBODIMENT OF THE INVENTION

To avoid the problems of the prior art [of] embodiment illustrated in FIG. 3, the present invention utilizes a valve system [is modified] as shown in FIG. 4. [The modifications include removal of shear valve 58 from pocket 60. In addition,] In FIG. 4 as compared to FIG. 3, the check valve 120 is [also] removed. After shear] from the valve pocket 123. When the shear valve 58 is removed from the valve pocket 60. All] all grease is removed from the O-ring seal 84 and the T-seal ring 66. The shear valve 58 is then lubricated with Baker Tubing Seal Grease Number 499-26 which is not reacting with the O-ring seal 84 or the T-seal 66 at elevated temperatures. The shear valve 58 is] and then replaced in the valve pocket 60 in the manner known in the prior art. [Pocket] The valve pocket 123 is then filled, preferably with water or other suitable substance, although it could be left unfilled. The shear valve 58 thus has a reciprocating valve element or valve body member 59 with seals 66 and 84 which prevent flow of fluid to the passage 71 (part of the passageway means) and stop means in the form of the shear pin 106 which prevent reciprocation of the reciprocating valve member prior to the application of a predetermined pressure difference.

[A modified] As shown in FIG. 4, a retainer housing 182' is [then] installed in the valve pocket 123. The [modified] retainer housing 182' includes a bore 200 of smaller diameter than the bore 178 [drilled coaxially through bore 178. Housing] The housing 182' [is] further [modified to include] includes a counter bore 202 coaxial with and of smaller diameter than the bore 200 and is formed by drilling or other means through the approximate center of a transverse groove 198. The disparity of diameters causes an downwardly, outwardly, sloping shoulder 204 to be formed between the bore 178 and the bore 200 and an downwardly, outwardly sloping shoulder 206 to be formed between the bore 200 and the bore 202. A ball 208 is located within the retainer housing 182' in close proximity to the opening of the bore 202 facing the bore 200. [Ball] The ball 208 is held against the shoulder 206 by a compressed spring 212. [Spring] The spring 212 is compressed by a rod 210 which contains an internal longitudinal fluid passageway 211 extending therethrough and opening at each end. [Rod] The rod 210 is inserted into the bore 178 by hammering or other means to force the rod 210 into the entry of the bore 200 where it is held by friction with the spring 212 and the ball 208 extending into the cocurrent bore 200 such that the ball 208 abuts the shoulder 206 and the rod 210 extends substantially into the shoulder 204 forming a check valve assembly 214.

The flow passage or port 56 is independent of the seals 84 and 66 for permitting flow of fluid from the interior or bore of the pipe member 11 to one side of the seal 66 and the retainer housing 100 is independent of the seals 84 and 66 for permitting the flow of fluid from the exterior of the valve collar 14 and the pipe member 11 to an oppositely facing surface of the seal 84.

The structure in the valve pocket 123 comprises means independent of the seals 84 and 66 for permitting the flow of fluid via passage 71 at the pressure exterior to the pipe member 11 and the valve collar 14 to the other side of each of the seals to equalize the pressure across the seals 66 and 84.

In operation, when the valve system of FIG. 4, is installed in the [member 10 of the first embodiment FIG. 4, when] valve collar 14 and lowered into the bore hole, [will cause] the pressure in the passageway 71 [to] will be approximately the same as the pressure in upper [pocket] chamber 65 of [bore] the valve pocket 60. This is effected by the check valve assembly 214 in the valve pocket 123. As pressure from the bore hole acts on the tubular pipe member [10] 11, and particularly on [modified] the retainer housing 182', fluid will flow from the counter bore 202 through the bore 200 to the bore 178, around the ball 208, through the passageway 211 in the hollow rod 210 and thence to the valve pocket 123, the port 127 and the passageway 71. This will permit the fluid in the valve pocket 123 to be maintained at the pressure approximately that surrounding the tubular member [10] 11 which is substantially the pressure in the upper [pocket] chamber 65. Accordingly, the differential pressure between the upper [pocket] chamber 65 in the valve pocket 60 and the pressure in the passageway 71 across the seal 84 will be very small. Further, the pressure at the port 56 will also initially be approximately that of the well bore so that the differential pressure across the seal 66 will be very small. In addition, as the pressure in port 56 [increases, and] is increased to shear pin 106 [shears, causing] and cause the valve body 59 to move [such that], the seal 66 moves to a position longitudinally above passageway 71[,] and the pressure in the valve pocket 123 will increase causing the ball 208 to seat on the shoulder 206 thereby stopping further fluid communication between the bore 200 and the bore 202. Therefore, the pressure in the passageway 71 [will] can continue to rise or increase causing the check valve 122 to unseat and permitting fluid flow to the passageway 137.

The [modified] port plug 182' is usually covered with Shell Darina Grease Number 2 or other suitable lubricant to prevent plugging of the check valve 214.

In addition, because multiple packers are usually run along a tubular string comprised of tubular pipe members [10] 11 and other tubular members, the seal diameters should be measured and an indication of such be made, such as on the valve collar 14. In this manner the packer with the smallest upper seal 84 area will be run closest to the bottom of the hole to minimize distortion caused by different areas between the seals 84 and 66 since the devices of the prior art always have a larger area for the seal 84 than for the seal 66.

Although the system described in detail above is most satisfactory and preferred, many variations in structure and method are possible.

Because many varying and different embodiments may be made within the scope of the inventive concept herein taught and because modifications may be made in accordance with the descriptive requirements of the

law, it should be understood that the details herein are to be interpreted as illustrative and not in a limiting sense.

What is claimed as invention is:

1. A tubular system for use in packing off a well bore, comprising:

a [hollow] tubular mandrel;

[a packer] an inflatable packing element attached to said mandrel at one end of said mandrel;

a valve collar mounted on the other end of said mandrel, the other end of said [packer] packing element being attached to said valve collar [and], said valve collar having a passageway means there-through, [and] said passageway means in said valve collar being in fluid communication with the interior of said [packer] packing element and the interior and exterior of said mandrel [by said passageway];

at least two valve pocket means in said passageway means in said valve collar;

[said passageway having enlarged portions in said collar;

a valve system mounted in said enlarged portions, said valve system including three valves;]

[the first of said valves being mounted in the first of said portions and having] first valve means in a first valve pocket means and including a reciprocating valve member and [a] stop means for preventing reciprocation of said reciprocating valve member from a first position to a second position prior to the application of at least a predetermined difference in pressure between one [side] end of said reciprocating valve member and the other [side,] end of said reciprocating valve member, said reciprocating valve member being located in said first position at one open end of a first part of said passageway means when said stop means prevents reciprocation and having at least two spaced apart seals [thereon] which are disposed so that the one open end is between said seals for preventing the flow of fluid from either end of said reciprocating valve member around the seals on the reciprocating valve member to said [first] one open end of the first part of said passageway [part] means;

said first valve pocket means being in fluid communication [with a second part of said passageway in fluid communication] with [said] the interior of said mandrel [on] and one side [of] of one seal on said reciprocating valve member;

said first valve pocket means being in fluid communication with [said] the exterior of said mandrel [on the other] and one side of the other seal on said reciprocating valve member; and

[the second of said valves having check means] second valve means in a second valve pocket means for permitting the flow of fluid in one direction from [said] the exterior of said mandrel to said one open end of the first part of said passageway [when the pressure exterior of said mandrel exceeds the pressure in said second portion] means between the seals;

said second [portion includes] valve pocket means including a first bore [opening] which opens to [said] the exterior of said mandrel and [said second] check valve [is] means located in said first bore;

said second valve [includes] pocket means having

a [head adapted to connect] *retainer housing connected* to said first bore,  
 a seal mounted on said [head] *retainer housing* and sealingly engaging the [walls] *wall* of said first bore and said [head] *retainer housing*,  
 [and] said check valve means [is] *being mounted in* said [head] *retainer housing*;  
 said check valve means [includes] *including*:  
 a second bore through said [head and] *retainer housing* substantially coaxial with said first bore;  
 a third bore through a portion of said [head] *retainer housing* substantially coaxial with said first and second bores, said third bore being of a greater diameter than said second bore and forming a shoulder therewith;  
 a ball adapted to seal against said shoulder; and mounting means for reciprocably mounting said ball in said third bore.

2. The system of claim 1 wherein said mounting means includes:  
 a rod, said rod being wedged into an end of said third bore;  
 a spring, said spring being mounted in said third bore and abutting said ball and one end of said rod.

3. The system of claim 2 wherein said rod has a hollow longitudinal fluid passage therethrough.

4. A tubular system for use in packing off a well bore, comprising:  
 a tubular mandrel;  
 an inflatable packing element attached to said mandrel at one end of said mandrel;  
 a valve collar mounted on the other end of said mandrel, the other end of said packing element being attached to said valve collar, said valve collar having a passageway means coupling the interior of said packing element and the exterior of said mandrel in fluid communication with the interior and exterior of said mandrel;  
 at least two valve pocket means in said passageway means in said valve collar;  
 first valve means in a first valve pocket means and including a reciprocating valve member and stop means for preventing reciprocation of said reciprocating valve member from a first position to a second position prior to the application of at least a predetermined difference in pressure between one end of said reciprocating valve member and the other end of said reciprocating valve member, said reciprocating member being located in said first position at one open end of a first part of said passageway means when said stop means

*prevents reciprocation and having at least two spaced apart seals which are disposed so that the one open end is between said seals for preventing the flow of fluid from either end of said reciprocating valve member around the seals on the reciprocating valve member to said one open end of said passageway means*;  
*said first valve pocket means being in fluid communication with a second part of said passageway means where said second part of said passageway means is in fluid communication with the interior of said mandrel and one side of one seal on said reciprocating valve member*;  
*said first valve pocket means being in fluid communication with the exterior of said mandrel on one side of the other seal on said reciprocating valve member; and second valve means in a second valve pocket means for permitting the flow of fluid in one direction from the exterior of said mandrel to said one open end of the first part of said passageway means between said seals*.

5. The system of claim 4, wherein said second valve means includes:  
 a retainer housing for enclosing said second valve pocket means, said retainer housing having an opening there-through;  
 a seal mounted on said retainer housing and sealingly engaging the wall of said second valve pocket means and said retainer housing; and  
 check valve means in the opening of said retainer housing.

6. The system of claims 4 or 5 wherein the said first part of said passageway means is filled with a liquid.

7. The system of claim 4 wherein;  
 there are at least three valve pocket means,  
 third valve means in a third valve pocket means,  
 said third valve means being in fluid communication by said passageway means (a) with said exterior of said mandrel (b) with said second valve pocket means and (c) with the interior of said packing element;  
 first seal means on said second valve means for preventing fluid communication between second valve pocket means and said exterior of said mandrel, and  
 said third valve means in said third valve pocket means having  
 second seal means for preventing fluid communication between said second pocket means and said interior of said packing element when the pressure in said interior of seal packing element is greater than the pressure in said second valve pocket means.

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