

[54] Title: HYDRAULIC RELEASE OIL TOOL

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[57] (see abstract next page)

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ABSTRACT

DESCRIPTION OF THE INVENTION

The present invention relates to tools used in drilling and completion of oil wells, and in particular, the invention relates to a tool that can be used to reduce friction between a liner and a well bore or to release a well string for its removal from a well bore.

In the drilling of oil wells, a drill string is used to drill a well hole. After the well hole is drilled, casing or liner is run across the produced interval of drill pipe, with the liner in place, water is pumped down through the drill pipe, through the liner and into the annulus between the liner and the casing in the well hole. This causes the liner to swell and to keep production fluids in the liner.

Some of the well holes are not completely vertical. They may have substantial horizontal components. These are substantial horizontal components and are called high angle, extended reach wells. Running a liner through high angle, extended reach well holes can be difficult because of friction between the liner and the well hole caused by the weight of the liner against the wall. Translational friction can be substantially reduced by rotating the liner while it is being run down the well hole.

The tools that run liners into well holes have not been able to rotate the liners while they were being run in, although the tools permit liner rotations during

coupling member. The locking assembly includes a body that contains a piston. The piston is a working piston attached to the body, primarily through outer pins, so that it does not move with respect to the body until it does a work string process that produces a force above a value sufficient to release the piston when outer pins are used to move the pins. When this pressure is reached, the piston moves in the body to a release position. Dogs extend in the body for radial movement between extended and retracted positions. Working means such as springs, bias the dogs towards their retracted position. Means on the piston keep the dogs in their extended position when the piston is in its working position. A radially outward extending bias on the piston radially provides their bias. The dogs in their extended position engage the coupling member through sockets so that the coupling member is rotationally and translationally attached to the cutting assembly. Means on the piston, such as a relief pin, the body, permits the dogs to retract when the piston moves to its retracted position. When the dogs retract the coupling member and the cutting assembly are no longer attached and can be used again. Ordinarily, the piston acts as a relief part between work string produced and ambient well pressure. When the piston moves to its release position, the part communicated the work string pressure to ambient pressure, lowering the work string pressure. This signals the completion of recovery of the

procedure is then applied immediately after the
the shooting phase. This happens either a liner super dart
has been forced into the liner to wipe around off the
middle of the liner. This super dart lands on a landing
roller and proceeds in a step sufficiently to clear the
5 phase. When the gun moves, the piston moves within the
body to clear the land from the back of the dogs. When
the dog breaks free of the land, the springs withdraw the
dogs from the magazine, releasing the water gun assembly
from the running head. A chamber of the head of the
10 dogs insures that water from the magazine exits
applied to the dogs by the running head through the
chambers, providing initial lubrication to clear the dog.

Sometimes, the procedure in the working string could
be increased enough to shift the piston and release the
15 dogs. For example, if the super dart does not land on
the landing roller, the string procedure cannot be raised
sufficiently. When this happens, a pump down dart with a
plug is pumped down the work string and lands on the
20 piston of a hydraulic release tool, sealing the
string at the point. With a seal at the piston, a lower
pressure is used than that the pressure otherwise
required because no area of the piston is exposed for the
pressure to act the other way to oppose clearing force.
30 Further, the bore through the piston has been cleared of
the seal, and working string pressure on the seal
area has increased the force on the piston from water
to clear the area.

The hydraulic release tool consists of a drill string release tool and a liner release tool. The release tool is adapted to hold the liner release tool only to adapt it for placement of the drill string through a drill string release sub.

5 In use, the drill string release adaptation works just like the liner release counterpart. Hydraulic pressure is introduced sufficiently to where the pins that suspended the piston, and the piston moves to clear a line from the base of dogs that engage sockets in a release sub. The release sub corresponds to the running
10 head. Springs act on the dogs to retract them from the sockets, and the drill string is separated from its stuck position. The separation portion of the drill string can then be pulled from the hole and the continuous clearing end of the string released from the hole through overhead techniques, as through zero and fishing.

There are other features, aspects and advantages of the present invention will become more apparent from the following description, appended claims and drawings.

20 BRIEF DESCRIPTION OF THE DRAWINGS

FIGURE 1 shows the hydraulic release tool of the present invention, especially suitable for liner run in, cementing, partly in half-section.

FIGURE 2 shows a hydraulic release tool especially suitable for a drill string release, again partly in half-section;

FIGURE 3 illustrates schematically the operation of the tool of the invention, and

FIGURE 4 shows a pump down port that may be used with the present invention to increase the pressure on the piston to move it and release the release tool in the event that conventional techniques do not work.

DESCRIPTION OF THE PREFERRED EMBODIMENT

With reference to FIGURE 1, a hydraulic release tool 10 of the invention is shown in an open position in use. Attached normally to a work string between a drill pipe and the liner, the tool has a shear cut setting assembly 11 and a coupling member in the form of a running head 12 that is adapted to 14. The shear cut setting assembly and the running head have clear through axial bores for working fluids used in drilling and completing a well.

The shear cut setting assembly has four radially extended lugs 15 that abut the shear cut assembly, or the running head 12, extended and release the tool when retracted. A spring 16 for each of these lugs biases its dog inward toward a longitudinal axis 17 of the assembly, out of the extended position shown in FIGURE 1, for the release of the running head. Each lug has a latched chamber 19.

Shear cut setting assembly 10 includes a body 18 and a piston 20 within the body. The piston can translate along axis 17 in the body in response to a

efficient differential hydraulic procedure return the piston
ends from a setting position to a release position. The
position of FIGURE 1 is the setting position. The piston
is retained in the setting position of FIGURE 1 by a
plurality of pins 23. The pins are retained
in a ring 24 and secured into the piston. Ring 24
abuts an interior shoulder 25 of body 18. An annular
internally biased 26 normally engage the interior ends of
pins 23 to extend the pins against the retracting force
of spring 22. Annular cheeks 27 and 28 on the upper
and downward ends of the pins reduce the diameter of the
piston at the cheeks. In this specification, words of
direction are with reference to the work string and well
bore.

Piston 20 has an upper face 29 normal to axis 17 of
the assembly in the communication with and acted upon by
hydraulic pressure in the work string that applies a
downward force on the piston to the right in FIGURE 1.

The piston also has a lower face 30 which sees
hydraulic pressure in the work string that applies an
upward force on the piston that tends to force the piston
to the left in FIGURE 1. The area of face 30 is larger
than the area of face 29 so that the net force on the
piston from the hydraulic pressure in the string is
downward. When the net pressure corresponds to the
downhole forces sufficient to shear the shear pins, the
pins shear. (This pressure correlates with the force
required to shear and is conventionally referred to as the

specifically to low production pressure.

A control seat 33 at one upper end of the pipe 31
side through the pipe, except a well for the plug seat
data of FIGURE 4, if one is necessary. A plurality of
longitudinal bleeders 34 in the wall of the portion
that defines the internal cavity 29 communicates liner
pressure with the pressure in the well between the
liner and the casing when circumstances are hydraulic
well. When the well is in the retracted position,
the bleeders close and communicate the liner and the
casing through an element that receives the seal.

Running head 37 has four seats 38. These seats
receive members 39 to attach the running head assembly
to the running head. Member 39 of the seat
permits axial movement of the dogs by the running head
to force the dogs to retract if the spring force is not
enough.

A top sub 35 of the well head assembly 37
connects to body 19 through threads 36 between top. Sub
35 has an internal thread 40 for connection to drill
pipe.

Running head 37 has a bladder built in sub 37 and a
release sub 41. The release sub has seat 34. Release
sub 44 has a pin 42 between sub 42 through a threaded
threaded connector 43. The pin 42 and connector
typically with a liner assembly.

Before proceeding with the detailed description
the operation of the hydraulic release well is shown.

in use with a well will be briefly described.

When drawn out working assembly 12 and running tool 14 attached together by dog 15 engaged in socket 24, the tool is moved by wire line, and running tool 14 turns a liner assembly. The liner assembly is run into a well hole while being turned. Accordingly, the friction between the wall of the well hole and the liner will be lower than it would be if the liner were not rotating. Because of the rotation, the problem that friction creates in large angle, extended reach well holes is eliminated, or at least substantially reduced.

When the liner has been run into the well hole and well cementing completed, hydraulic pressure within the work string is increased sufficiently to fail shear pin 22, and the piston moves downward. When the piston moves downward, it presents relief 27 to the case of dog 15. The dog, not being restricted by land 28, responds to the radially inward force of spring 16 and moves out of socket 24 and out of engagement with the running tool 14. The movement of the dog 15, and member 17, against the angle of socket 24, the dog and member 17 retreats because of the resulting radial inward force. The result is that assembly 12 that part of the running tool. The entire working assembly is left in with the wire line and can be pulled out of the hole.

Completing the detailed description of the invention shown in FIGURE 1, body 10 has a lower portion 40 and 46. The lower end has an internal shoulder 42 at the

15. A flexible pipe 52 extends from the rear 51 and provides coupling of the chain and driving assembly with another work element component. The body 48 is used for loading and discharging section 53 or a chamber 54. This section carries a piston 55 that passes 15 through the body. An external flange 56 extends radially outward from the interior end of intermediate section 53. The chamber has a right angle loading face 60 and a circular loading face 62; the latter keeps debris and other foreign matter from between the hydraulic release 20 seal and the interior of the running head. The trailing face of the chamber transcends into a trailing end 64 of the body. A pair of O-rings 66 seal the interface between the body and release sub 44 of running head 14. Thus 60 opens the upper end of chamber sub 44 when the chain sub assembly and running head are connected.

Piston 55 has an external flange 70 at its upper end. This flange mounts O-rings 72 that seal the piston and body bore wall so that hydraulic pressure does not pass the flange into an annular relief 74 behind the flange. This relief annular ring 74 and relief pin 76. Once a seal relief pin 76 seals the pressure within the annular relief so that if the downhole pressure externally of the tool, when flange 70 passes the relief pin, the work string pressure drops because fluid passes through the port to the outside of the tool. This drop in pressure signals the operator that the chain sub assembly and running head are con-

finger threaded together and made stationary. A pin 25 is between the work string and the annular member 24. The outer casing 20 is rotated along with the landing of a liner pipe in a casing collar.)

5 A shoulder 28 on an internal flange 29 at the bottom end of relief 20 provides a stop for the piston by engaging the landing end of land 26. O-rings 31 on the interior axial surface of flange 29 seal against a surface of the piston to keep work string fluid out of relief 29. A spring keeper 32 in recess 34 at the top of spring 30 holds the spring in place. Each taper 33 is interfered with the wall of the recess.

A plurality of seal grooves 36 complete the isolation between the pin 35 and body 18. The seal grooves prevent interference on threaded connection 37. The threads of the threaded connection, however, take the axial load.

FIGURE 2 shows the present invention adapted as a pull string release tool 100.

20 The pull string release tool 100 is a member of a lifting assembly 101 that has a body 112 and a piston 114 received in the body. These components move longitudinally with 113. A sub 116 connects a body 117 of a threaded and pin screw connection 118 to the piston of the previously described component. A release pin 119 is longitudinal with 118. It also has a pin internal shoulder 120.

The body 117 provides 121 on its external surface for

design of the piston and the cylinder. The piston and the cylinder are connected by a pin and the pin is secured by a nut and washer.

FIG. 1 is a cross-sectional view of the piston and the cylinder. The piston is shown in the left hand position and the cylinder is shown in the right hand position. The piston is connected to the cylinder by a pin and the pin is secured by a nut and washer. The piston is shown in the left hand position and the cylinder is shown in the right hand position. The piston is connected to the cylinder by a pin and the pin is secured by a nut and washer. The piston is shown in the left hand position and the cylinder is shown in the right hand position. The piston is connected to the cylinder by a pin and the pin is secured by a nut and washer.

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5 1000 psi. The relief valve is set at 1000 psi. When the pressure reaches 1000 psi, the valve opens and the fluid flows out of the well. When the pressure reaches 1000 psi, the valve opens and the fluid flows out of the well. When the pressure reaches 1000 psi, the valve opens and the fluid flows out of the well.

As before, the body has an external flange 156 that has a right angle face 158 that engages the end of the release tool 160. The trailing end of the flange 156 is used, as shown, to keep debris out of the connection between the hydraulic release tool and the interior of the running head.

The piston and body of the shear cut assembly also seal as an interface at their leading end by the flange 160 seated in the internal flange 162 of the body engaging a tapered surface at the leading end of the piston.

Release tool 170 has an external thread 183 for a drill pipe connection with the shear cut assembly 10 pulled from the hole. The leading end of the release tool has an external thread 184 also for a drill pipe connection.

With reference to FIGURE 3, a typical application of the embodiment of the invention shown in FIGURE 1 is illustrated. In FIGURE 3, a string of drill pipe 190 connects directly to the hydraulic release tool 10. With reference to FIGURE 1, that connection would be through an external thread on the end of the drill pipe and internal thread 4a. The hydraulic release tool, in turn,

connects vertically to other components in the well string.
The components shown are a casing head 192, a liner hanger
194, a liner hanger 196, liner 198, a landing
collar 199, a float collar 193, and a float shoe 194.
5 Depending on the application, the safety joint of the liner
hanger can be eliminated. A top drive or (power swivel)
195 rotates the work string. A rotating cement head 197
isolates cement slurry from the drill pipe string.

FIGURE 4 shows a pump down wiper dart used in the
10 event that hydraulic release tool does not function
because of an inability to increase work string pressure
sufficiently to shear the shear pins. A wiper dart 201
has a head 204 and a plurality of radial rubber lips 202
203 within the head. The wiper aligns the dart with the
15 working string. Forward of the dart is conventional ball
209 spacer from the release of the wiper by an external
agent 205 can act against seat 21 of the setting
assembly shown in FIGURE 1.

The application of the liner release tool will be
20 described with reference to FIGURES 3 and 4. A liner
198 is run into a well hole while rotating drill pipe 195
through top drive or power swivel 195. Rotation of the
drill pipe rotates the hydraulic release tool. Because
the hydraulic release couples to the liner, the liner
25 also rotates. The string saves yawhole as it rotates.
Rotary motion, the friction that would otherwise exist
between the liner and the well hole wall is eliminated
and extends reach of the liner. The liner is properly

located in the drill pipe, in the case of a downer. The
liner is to be used to remove cement during cementing work. It
power of top drive or power shovel.

Cement is pumped in the interior of the drill pipe
string and introduced liner to fill them. Then a drill
5 pipe wiper sock is released into the drill pipe. A fluid
under pressure behind the drill pipe wiper sock forces
the sock through the drill pipe string, wiping the
cement from the interior of the drill pipe. When the
10 drill pipe wiper sock reaches the bottom of the drill
pipe string and the top of the liner, it latches into a
larger diameter wiper plug, and the combination of the
two plugs is pushed through the liner to wipe and
displace the cement from the interior wall. The liner
15 wiper plug finally bumps into the landing collar and
slips.

Then the liner wiper plug bumps into the landing
collar, pressure in the work string is increased, say to
2,000 psi, to stretch liner hanger 186, which engages the
20 top of the liner with the inside of the previously set
landing so that the weight of the liner is taken by the
well where the hanger engaged it. Pressure in the work
string is then increased, say to 3,200 psi, to shear the
liner out setting assembly from the running head. The
70 drill pipe and releaser hook out section is then
then pulled from the hole.

In the event that the wiper plug does not load
on the landing collar so that work string pressure

Accordingly, plug 190 is moved 192 in direction 194 and the
string is released and moving assembly from the 190, 192
downward. In accordance with reference to FIGURE 6, 190 and 192
down 200 is pushed down the work string. Head 204 and
rubber expand 206 pass through the wire of the piston,
and ball plug 208 lands on seat 33, sealing the work
string downstream of the head. With the seal, hydraulic
pressure upstream of the ball is not opposed by pressure
downstream of the ball or seat 33. Further, the area of
the piston wire through the ball seal work string
pressure, increasing the shearing force on the shear
pins. Accordingly, the shear pins shear at a lower
pressure, and the piston moves for the retraction of the
logs.

With ball 208 on seat 33, back pressure valve 19
float collar 192 and float shoe 194 force the ball into a
tighter seal on the seat when an attempt is made to pull
the logging assembly from the running head. At this
time, elements 35 are radially inwards of springs 36
because the piston is in the retracted position.
Accordingly, the pressure downstream bleed down through
the bladder valve 35 across the interface between lands 17
and the adjacent wall of the body and out window 76.

The drill string valve 35 operates in the same
manner as the line valve released and is
designed for use for a different purpose. The drill
string valve is used to provide a downstream way for
upsetting the drill string in the event that its lower

to slide

1. A hydraulic release tool comprising:

(a) a setting assembly including:

(i) a body having a longitudinal axis and a

longitudinal bore along the axis;

(ii) a piston translatable in the bore of the body

between a setting position and a release position,

the piston also having a longitudinal bore and

differential area to effect a net force on the

piston in response to hydraulic pressure in the

bore of the body that tends to move the piston

from the setting position to the release

position;

(iii) a plurality of dogs in the work substantially

radially of the axis between a retracted position

close to the axis and an extended position within

the body and radially of the body;

(iv) driving means for biasing the dogs toward the

extended position;

(v) means on the piston engaging the base of the

dogs to extend the dogs outside the body when the

piston is in the setting position;

(vi) means on the piston to disengage the base of

the dogs so that the dogs move to their retracted

position in response to the dog biasing means when

the piston moves to the release position; and

(vii) means between the piston and the body to seal

the piston in the setting position until a

predetermined hydraulic pressure exists in the body 10 and to release the piston 12 when the hydraulic pressure exceeds the predetermined pressure so that the piston moves to its release position;

5 (iv) a coupling member for coupling to drill string or pitman product or components, the coupling member having a plurality of sockets for receiving the seat in its extended position and attaching the coupling member to the setting assembly; whereby the setting assembly couples to the coupling member by engagement of the dogs in the sockets and releases from and disengages the coupling member upon movement of the piston from its setting to its release position in response to a pressure in the body in excess of the 10 predetermined pressure; and

(v) a relief seat that enters the body into its seat and seals to prevent communication between the tool joint and the bore of the body axially of the piston when the piston is in the setting position and to establish such communication upon movement of the piston from the setting position to the release position and reduces the pressure in the body axially of and piston, whereby the reduction of the pressure signals that the tool has released the 15 coupling member.

2 The hydraulic circuit of claim 1 wherein the work on the piston is engaging the rear of the relief

includes a land on the piston positioned as about the base of the dogs in the setting position of the piston.

2. The hydraulic release tool of claim 2 wherein the means on the piston to disengage the base of the dogs includes a relief of the piston behind the land that passes under the base of the dogs when the piston moves from its setting position to its release position.

3. The hydraulic release tool of claim 3 wherein the sliding means includes individual springs between the dogs and the body.

4. The hydraulic release tool of claim 4 wherein the sliding means includes a plurality of shear pins between the body and the piston that fail at the pressure above the predetermined pressure.

5. The hydraulic release tool of claim 1 wherein:
(a) the means on the piston for engaging the base of the dogs includes a land on the piston positioned as about the base of the dogs in the setting position of the piston.

(b) the means on the piston to disengage the base of the dogs includes a relief of the piston behind the land that passes under the base of the dogs when the piston moves from its setting position to its release position.

6. The hydraulic release tool of claim 6 wherein the sliding means includes a plurality of shear pins

attracting the body to the piston that will be in pressure above the predetermined pressure.

6. The hydraulic release tool of claim 7 wherein the biasing means includes individual springs between the dogs and the body and each dog has a chamfered end at its radially outward end to apply a radial force toward the longitudinal axis by engagement of such end with the wall of the socket in the coupling member.

9. The hydraulic release tool of claim 8 wherein the coupling member comprises a running end, the running end having means to attach to a liner to permit the rotation and translation of the liner during liner run in and rotation of the liner during assembly.

10. The hydraulic release tool of claim 1 wherein:
(a) the means on the piston for engaging the base of the dogs includes a land on the piston positioned to abut the base of the dogs in the setting position of the piston; and
(b) The means on the piston to disengage the base of the dogs includes a relief of the piston behind the land such that, under the force of the dogs when the piston moves from the setting position to its release position.

11. The hydraulic release tool of claim 10 wherein:
(a) the biasing means includes individual springs between the dogs and the body; and
(b) the holding means includes a plurality of shear pins

between the body and the piston that fail at the pressure above the predetermined pressure.

12. The hydraulic release tool of claim 1 wherein the coupling member comprises a release end, the release end having means to connect it to a drill pipe, whereby the hydraulic release tool placed in a drill string separates the string upon release of the release end by the setting assembly.

13. The hydraulic release tool of claim 12 wherein:

- 10 (a) the means on the piston for engaging the base of the dogs includes a land on the piston positioned to abut the base of the dogs in the setting position of the piston; and
- (b) the means on the piston to disengage the base of the dogs includes a relief of the piston behind the land that passes under the base of the dogs when the piston moves from its setting position to its release position.
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14. The hydraulic release tool of claim 13 wherein:

- 20 (a) the holding means includes individual springs between the dogs and the body; and
- (b) the holding means includes a plurality of shear pins between the body and the piston that fail at the pressure above the predetermined pressure.

25 15. A hydraulic release tool comprising:

- (a) a setting assembly including:
- (i) a body having a longitudinal axis and a

longitudinal axis along the axis;

(iii) a piston translatable in the bore of the body between a setting position and a release position, the piston also having a longitudinal bore and differential means to effect a net force on the piston in response to hydraulic pressure in the bore of the body that tends to move the piston from the setting position to the release position;

(iv) a plurality of dogs in the body extendable radially of the axis between a retracted position close to the axis and an extended position further from the axis and outwardly of the body;

(v) dogging means for biasing the dogs toward the axis from the body;

(vi) means on the piston engaging the base of the dogs to extend the dogs outside the body when the piston is in the setting position;

(vii) means on the piston to disengage the base of the dogs so that the dogs move to their retracted position in response to the dog biasing means when the piston moves into its release position; and

(viii) means between the piston and the body to hold the piston in the setting position until a predetermined hydraulic pressure exists in the body bore and to release the piston when the hydraulic pressure exceeds the predetermined pressure so that the piston moves to its release position;

15 (14) a coupling member for coupling to a well string, the
perforation production components, and coupling member
having a plurality of contacts for receiving the dogs
in their extended position and attaching the coupling
member to the setting assembly;

20 (15) a relief port through the body and into the bore and
means to prevent communication between the relief
port and the bore of the body axially of the piston,
the port being in a position where, on movement of the
piston from the setting position to the release
position, the relief port reduces the pressure in the
body bore axially of the piston, thereby the reduction
of the pressure signals that the tool has released the
coupling member;

25 (16) a means of running in a liner in a well comprising
of steps of:

30 (a) a work string, attaching the liner to the running
head of a hydraulic setting and release tool that has
a setting assembly of a body having an axial bore, a
piston having an axial bore, the piston being in the
bore of the body, and a plurality of radially
adjustable dogs, the piston responding to a
35 predetermined axial force on the piston produced by
hydraulic pressure to move from a setting position to
a release position to retract the dogs from engagement
with the running head;

40 (b) running liner into position in the well hole by
turning it while moving it through the hole by force

applied to exert the pull,
(c) after the liner is in a desired position in the well hole, releasing the running head from the setting assembly by increasing the pressure in the work string to create the predetermined force to move the piston to the released position, retract the dogs, and disengage the running head; and
(d) relieving the line work pressure in the work string through a relief port in the body after disengaging the running head to signal the disengagement of the running head.

17. The method of running in a liner claimed in claim 17 including the step, before the releasing step of cementing the liner in the well hole under the liner master at desired position, while rotating the liner through the well.

18. The method of running in a liner claimed in claim 17 wherein the step of relieving the increased pressure includes the step of venting the pressure in the work string with movement of the piston to signal release of the running head.

19. The method of running in a liner claimed in claim 17 including in the releasing step landing a sealing disk on the piston to seal the piston's axial bore and increase the net down hole force to create the predetermined force at a lower work string pressure than is necessary without the disk.

20. The method of running in a liner disclosed in claim 19 wherein the sealing dart includes a bell mounted on the shaft and the piston has a seat facing uphole, the sealing step including engaging the bell on the piston seat

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21. A release oil tool having a coupling member and setting assembly; the setting assembly moving or attaching means, the tool comprising:

(a) a coupling member for coupling to the setting assembly, the coupling member having means for mating with the setting means;

(b) a setting assembly including:

(i) a body having a longitudinal axis and a longitudinal bore substantially along the axis;

(ii) means for releasably attaching the body to the coupling member's setting means; and

(iii) means for relieving pressure in the body bore upstream of the outlet of the attaching means from the setting means, wherein said means for relieving includes a relief port from the body bore to the outside of the body.

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22. The tool of claim 21 wherein the means for attaching includes a piston translatable longitudinally in the body bore from an attachment position to a detachment position, the piston being in the attachment position when the tool is attached to the coupling member.

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23. The tool of claim 22 wherein the piston has a

substantially longitudinally and in a downward direction from the body bore.

24. The tool of claim 21 wherein the attaching means is movable from the engagement position to the detachment position in response to a change in hydraulic pressure in the body bore.

25. A method of hydraulically releasing a coupling member from a setting assembly in a well comprising:

- (a) in a work string, attaching a coupling member to a setting assembly; (b) providing a means for hydraulically actuating the setting assembly to the coupling member and for releasing the coupling member in response to a change in hydraulic pressure; (c) running the setting assembly down a well; (d) increasing the hydraulic pressure in the work string to release the attachment between the coupling member and the setting assembly; and (e) relieving the increase in hydraulic pressure in the work string through a relief port to signal the release of the coupling member.

26. The method of claim 25 wherein the step of relieving the increase in hydraulic pressure includes driving a piston within the body bore in response to the pressure increase to uncover a relief port opening in the body bore.

27. A method of lining in a well in a well comprising:
(a) in a work string, attaching the liner to the running

means of a setting and releasing mechanism, the tool
having an axial part and means for hydraulically
attaching the liner to the setting tool, the means
releasing the liner in response to an increase in
5 hydraulic pressure;

- (b) running the liner into position in the well;
- (c) after the liner is in a desired position in the
well, releasing the setting tool from the setting
assembly by increasing the hydraulic pressure in the
10 work string, and
- (d) relieving the increase in hydraulic pressure in the
work string through a relief port after releasing the
liner to signal the release of the liner.

25. The method of claim 23 wherein the step of relieving
the increase in hydraulic pressure includes driving a
15 piston within the body member in response to the pressure
increase to uncover a relief port opening in the body
member.

26. A hydraulic elastic tool having a coupling member
and a setting assembly, the setting assembly having an
20 attaching means, the tool comprising:

- (a) a coupling member for coupling to the setting
assembly, the coupling member having means for mating
with the attaching means; and
- (b) a setting assembly including:
 - (i) a body having a longitudinal axis and a
longitudinal bore along the axis;
 - (ii) means for attaching the body to the coupling

member is sliding means, and means for moving said sliding member from an attachment position to a detachment position, the attaching means engaging the coupling assembly mating means to attach and coupling and sliding assembled in the attachment position;

4. (iii) means for allowing the attaching means to move from the attachment position to the detachment position in response to change in hydraulic pressure in the body bore; and

5. (iv) ported means for relieving pressure in the body bore to outside the body upon release of the attaching means.

26. The tool of claim 29 wherein the allowing means allows the attaching means to move from the attachment position in response to an increase in hydraulic pressure in the body bore.

27. The tool of claim 29 wherein the attaching means includes a plurality of dogs extendable radially of the body axis.

28. The tool of claim 31 wherein the mating means includes a plurality of sockets for receiving the dogs when the dogs are radially extended to the attachment position.

29. The tool of claim 32 wherein the sockets include side walls engaged by the dogs for transmitting motion rotational about the longitudinal axis from the setting

assembly or the coupling assembly.

1A. The tool of claim 2A wherein the setting means includes means for transmitting motion, rotational about the longitudinal axis from the setting assembly to the coupling assembly.

1B. The tool of claim 2A wherein the following means is means for producing a variable along the longitudinal axis in the body bore between a setting position and a release position, the piston having a first surface of engaging the attaching means engaging the attaching means in the attached position when the piston is in the setting position and a second surface not engaging the attaching means when the piston is in the release position.

1C. The tool of claim 3B including means for holding the piston in the setting position until a predetermined hydraulic pressure exists in the body bore and for releasing the piston when the hydraulic pressure exceeds the predetermined pressure.

1D. The tool of claim 3B wherein the holding means includes a plurality of check pins between the body and the piston that fail when the hydraulic pressure exceeds the predetermined pressure.

1E. The tool of claim 3B wherein the piston includes a landing seat for receiving and sealing against a dart and that when a landing dart is received the hydraulic

increased in the body and the increased

30. The tool of claim 27 including a landing seat in the body for receiving and sealing against a dart, the landing seat being positioned so that when a landing dart is received, the hydraulic pressure in the body bore is increased.

40. The tool of claim 29 including means for biasing the setting means toward the detachment position.

41. The tool of claim 40 wherein the biasing means includes a spring.

42. The tool of claim 29 wherein the coupling member comprises a running head, the running head having means to attach to a liner to permit the rotation and translation of the liner during liner run in and rotation of the liner during cementing.

43. The tool of claim 29 wherein the coupling member comprises a release sub, the release sub having means to connect it to a drill pipe, wherein the hydraulic release tool placed in a drill string separates the string upon release of the release sub by the setting assembly.

44. A hydraulic release tool having a coupling member and a setting assembly, the setting assembly having an attachment means, the tool comprising:

(a) a coupling member for coupling to the setting assembly, the coupling member having means for mating with the attaching means; and

21. a waiting assembly including:

(i) a body having a longitudinal axis and a longitudinal bore along the axis;

(ii) means for attaching the body to the coupling member's waiting means, the attaching means being moveable from an attachment position to a detachment position, the attaching means engaging the coupling member's waiting means to attach the coupling and detaching the coupling in the detachment position;

(iii) means for allowing the attaching means to move from the attachment position to the detachment position in response to changes in hydraulic pressure in the body bore wherein the allowing means includes an upper face in the body bore directed away from the coupling member and a lower surface face in the body bore directed toward the coupling member.

22. A hydraulic cylinder tool having a coupling member and a waiting assembly, the waiting assembly having an attaching means, the tool comprising:

(a) a coupling member for coupling to the waiting assembly, the coupling member having means for mating with the attaching means; and

(b) a waiting assembly including:

(i) a body having a longitudinal axis and a longitudinal bore along the axis;

(ii) means for attaching the body to the coupling

member's moving means the attaching means being
movable from an attachment position to a
detachment position, the attaching means engaging
the coupling assembly when moving toward the
coupling and disengaging assembly in the detachment
position;

(11) means for allowing the attaching means to move
from the attachment position to the detachment
position in response to changes in hydraulic
pressure in the body bore, wherein the following
means included: a piston movable along the
longitudinal axis in the body bore between a
locking position and a release position, the
piston having a first surface for engaging the
attachment means and urging the attaching means to
the attachment position when the piston is in the
locking position and a second surface not engaging
the attachment means when the piston is in the
release position and wherein the piston includes a
bottom face or surface, used for urging the first
surface into engagement with the attaching means
in response to hydraulic pressure in the body bore
and an upper face in the body bore for urging the
first surface away from engagement with the
attaching means in response to hydraulic pressure
in the body bore.

(12) The tool of claim 11 wherein the upper face has a
greater surface area than the bottom face.

17. A hydraulic release lock having a coupling member
and a setting assembly, the setting assembly including an
attaching means, the lock comprising:
(a) a coupling member for coupling to the setting
assembly, the coupling member having means for mating
with the attaching means; and
(b) a setting assembly including:
(i) a body having a longitudinal axis and a
longitudinal bore along the axis;
10 (ii) means for attaching the body to the coupling
member having means, the attaching means being
movable from an attachment position to a
detachment position, the attaching means engaging
the coupling assembly having means to attach the
15 coupling and setting assemblies in the attachment
position;
(iii) means for allowing the attaching means to move
from the attachment position to the detachment
position in response to changes in hydraulic
20 pressure in the body bore, wherein the following
means includes a piston translatable along the
longitudinal axis in the body bore between a
locking position and a release position, the
piston having a first surface for engaging the
25 attaching means and urging the attaching means to
the attachment position when the piston is in the
locking position and a second surface not engaging
the attaching means when the piston is in the

release position and wherein said relief part includes a hydraulic pressure relief part which extends outside the body into the body bore and means for preventing fluid communication between the relief part and the body bore when the piston is in the release position.

48. The relief part of claim 47 wherein the means for preventing and allowing fluid communication includes a flange on the piston for preventing or allowing fluid flow between the piston and the body at one end of the relief part in response to the position of the piston.

49. A hydraulic release tool comprising:

(i) setting assembly including

(a) a body having a longitudinal axis and a longitudinal bore substantially along the axis;

(b) a piston translatable in the body bore in response to changes in hydraulic pressure therein between a setting position and a release position, the piston having a substantially longitudinal bore and faces with differential areas to effect a net force on the piston in response to hydraulic pressure in the body bore that moves the piston from the setting position to the release position;

(c) a plurality of dogs in the body extendable radially of the axis between a retracted position close to the axis and an extended position further from the axis and laterally of the body;

(d) a first piston surface engaging the dogs for

48. The tool of claim 47 wherein said piston is in the retracted position when the holding means is in the setting position;
49. A hydraulic piston in fluid communication with the body for allowing the dogs to move to their retracted position when the piston is in its retracted position;
50. A relief valve between the piston and the body for holding the piston in the setting position until a predetermined hydraulic pressure exists in the body annulus; and
51. A relief port means for relieving pressure in the body from the outside the body upon the release of the holding means; and
52. A coupling member for coupling the setting assembly to a hydraulic cylinder, the coupling member having a plurality of sockets for receiving the dogs when in their extended position.
53. The tool of claim 49 including means for biasing the dogs toward the axis and into the body.
54. The hydraulic release tool of claim 50 wherein the biasing means includes individual springs between the dogs and the body.
55. The hydraulic release tool of claim 50 wherein each dog has a shoulder and at its radially outward end to apply a radial force toward the longitudinal axis by engagement of said end with the wall of the socket in the coupling member.
56. The hydraulic release tool of claim 55 wherein the

first piston member includes a land on the piston positioned to abut the edge in the setting position of the piston.

54. The hydraulic release tool of claim 53 wherein the second piston surface includes a relief behind the land that passes under the dog when the piston moves from the setting position to the released position.

55. The hydraulic release tool of claim 47 wherein the holding means includes a plurality of shear pins inserted in the body and land portion that fall when the hydraulic pressure exceeds the predetermined pressure.

56. A method of running a pipe in a well comprising the steps of:

(a) lowering the pipe to a setting tool or a tool having a substantially axial bore, a relief part, and means for responding to hydraulic pressure;

(b) running the pipe to a position in the well moving

(c) through the hole in, force applied through the tool;

(d) after the pipe is in a position in the well, releasing

the running tool by increasing the hydraulic pressure in the pipe; and

(e) relieving said increased hydraulic pressure by causing said bore through said relief part.

57. A hydraulic release tool comprising:

(a) a setting assembly including:

(i) a body having a longitudinal axis and a substantially longitudinal inner bore and

(f) a plurality of dogs or similar detent means

adapted to hold the attachment in position and a retracted position;

(g) means for allowing the dogs to move from the attachment position to the retracted position, including a piston translatable along the longitudinal axis wherein the upper face of said piston has a greater surface area than its bottom face;

(h) a coupling member for coupling the actuating means to the piston, the coupling member having a plurality of sockets for receiving the dogs when the dogs are in their retracted position;

50 The tool of claim 57 wherein the allowing means responds to hydraulic pressure in the body bore.

55 The tool of claim 58 wherein the allowing means allows the dogs to move from the attachment position to the retracted position upon or in the event the hydraulic pressure in the body bore

60 (i) The tool of claim 59 including means for preventing the allowing means from allowing the dogs to move to the retracted position until the hydraulic pressure in the body bore exceeds a predetermined pressure.

65 The tool of claim 59 wherein the upper face has a greater surface area than the bottom face.

68 The tool of claim 59 including a locking means in the

body 300 for allowing and sealing up said cavity, the
locking mechanism being positive and so that when a locking device
is recovered, the hydraulic pressure in the body 300 is
increased.

5 47. The tool of claim 37 including a hydraulic pressure
relief port 400 operable for moving the body 300 forward and
back for preventing fluid communication between the
relief port and the body bore when the piston is in the
locking position and allowing fluid communication between
10 the relief port and the body bore when the piston is in
the release position.

48. The tool of claim 47 wherein the means for
preventing and allowing fluid communication includes a
flange on the piston for preventing and allowing fluid
15 flow between the piston and the end bore in the relief
port in response to the position of the piston.

49. The tool of claim 37 including means for moving the
body toward the detachment position.

50. The tool of claim 49 wherein the moving means
20 includes a spring.

51. The tool of claim 37 wherein the coupling member
comprises a rotating head, the rotating head having means
to attach to a liner to prevent the rotation and
translation of the liner during liner run in and rotation
25 of the liner during retraction.

52. The tool of claim 37 wherein the coupling member

comprises a release arm, the release arm being means to connect it to a small pipe, whereby the hydraulic release arm placed in a small spring operated cylinder will release of the release arm by the setting assembly.

15 (C). A method of releasing a coupling member from a setting assembly in a well comprising:

(a) in a well string, attaching a coupling member to a setting assembly, said well string with a bore substantially along its longitudinal axis, a plurality of extendable dogs for engaging the coupling member, the dogs being retractable to release the coupling member, and a piston operable in the bore to alternately extend and allow retraction of the dogs;

(b) running the coupling member into position in the well by force applied through the well; and

(c) applying hydraulic pressure in substantially equal amounts to larger and smaller oppositely directed end faces of the piston in an amount sufficient to translate the piston in the direction of the smaller end face and allow retraction of the dogs.

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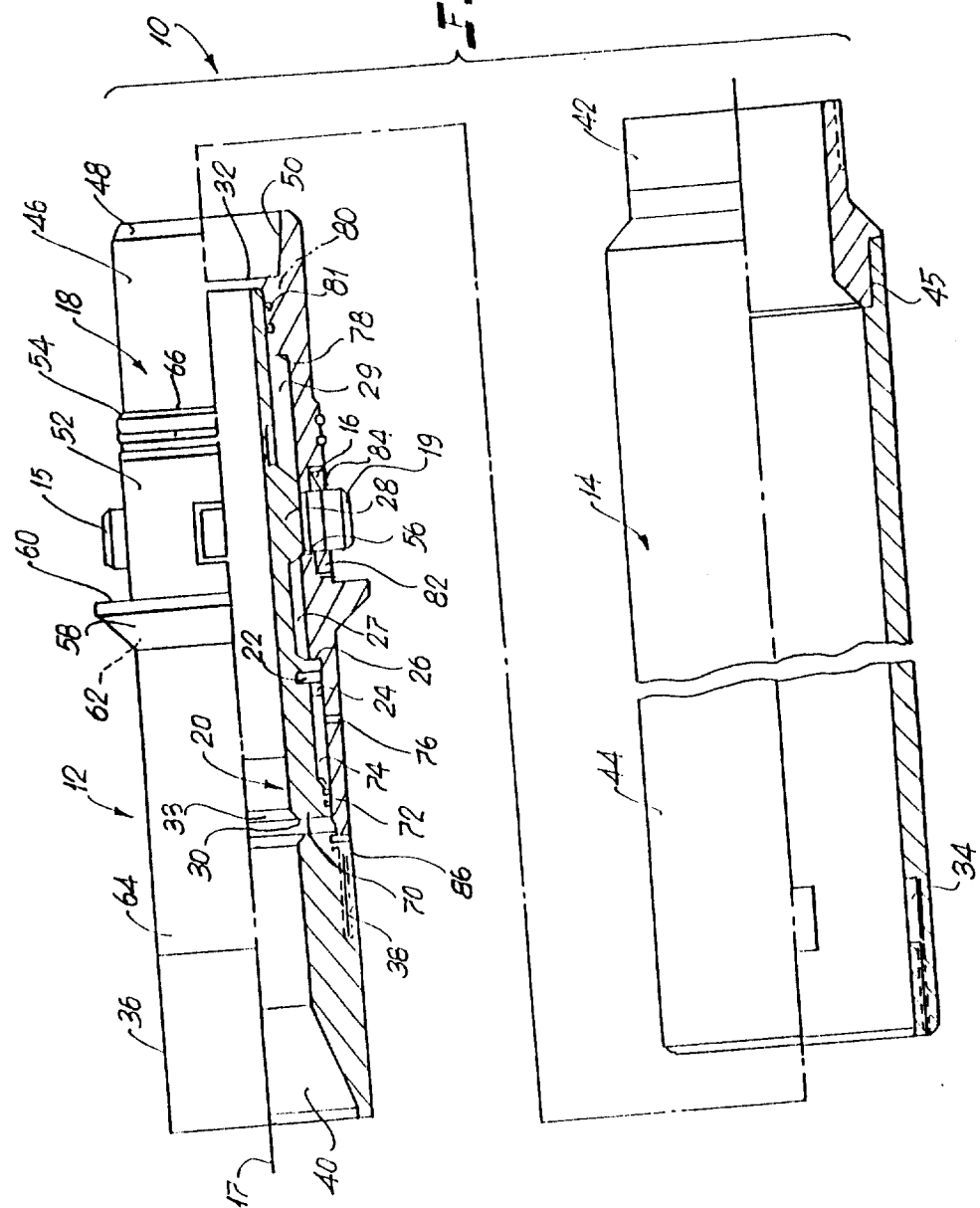
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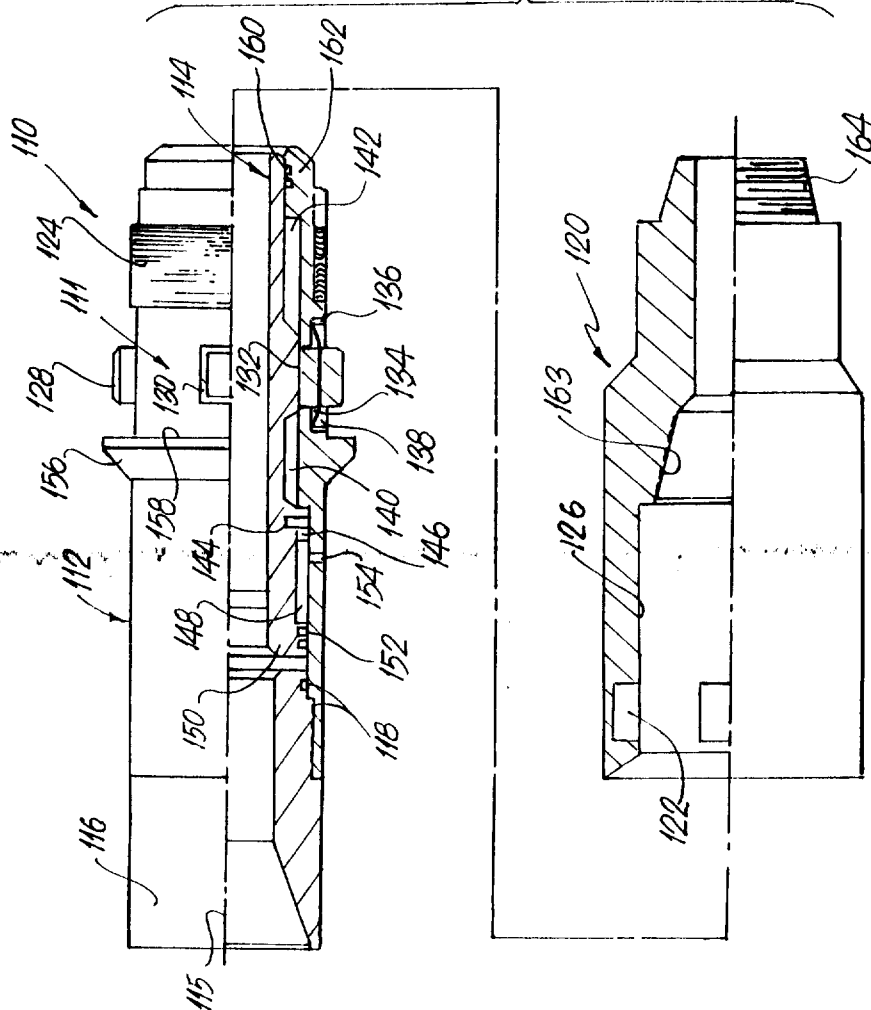


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FIG. 3.

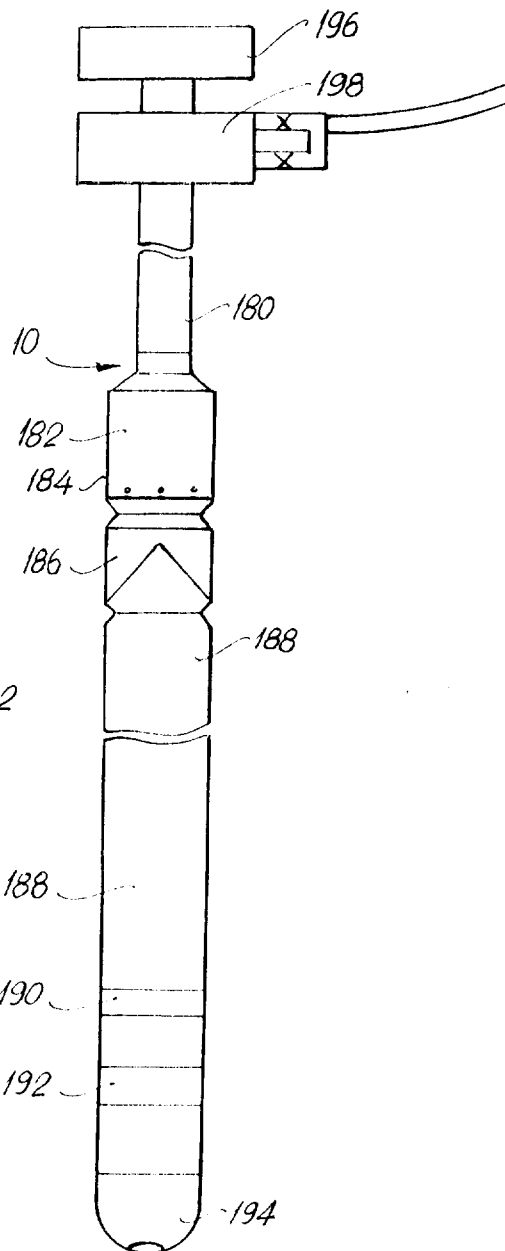
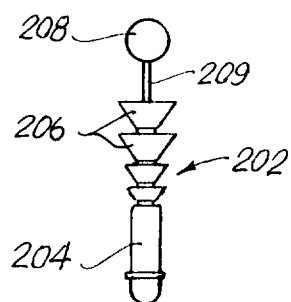


FIG. 4.



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