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Turk

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(54) **VENT WINDOW ACTUATOR**

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(52) **U.S. Cl.** **74/425; 74/416; 49/324**

(58) **Field of Search** **74/425, 416; 49/324**

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(57) **ABSTRACT**

An actuator for a vent window of a motor vehicle wherein the drive mechanism interconnecting the output shaft of the motor and the pivot arm controlling the vent window comprises a first worm driven by the motor output shaft, a first worm wheel driven by the first worm, a second worm driven by the second worm in driving the pivot arm. The first worm wheel is formed of a plastic material and the actuator further includes a coil spring mounted on the housing proximate the second worm wheel and operative to assist the second worm wheel in generating a sealing force sufficient to positively seal the vent window against a window seal of the motor vehicle.

8 Claims, 4 Drawing Sheets

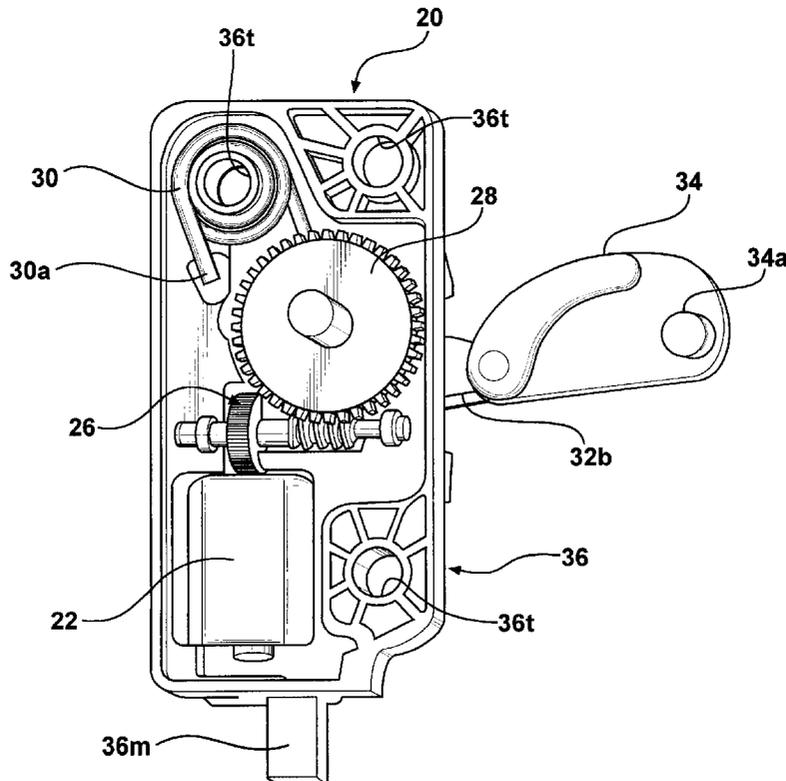


FIG - 1

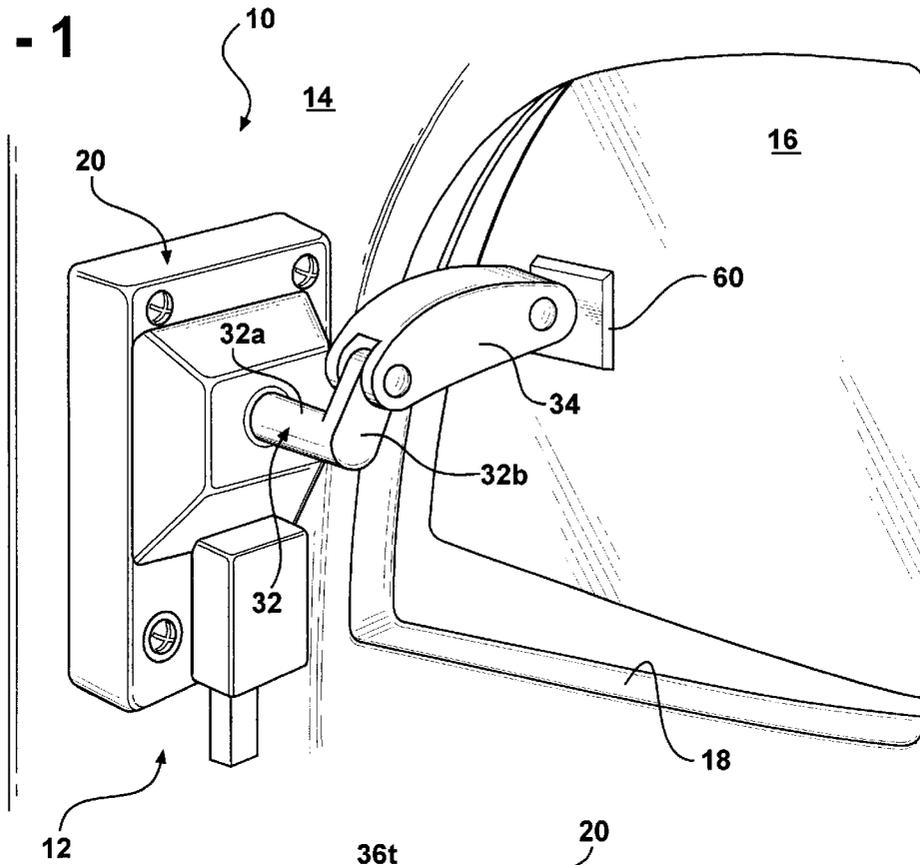


FIG - 2

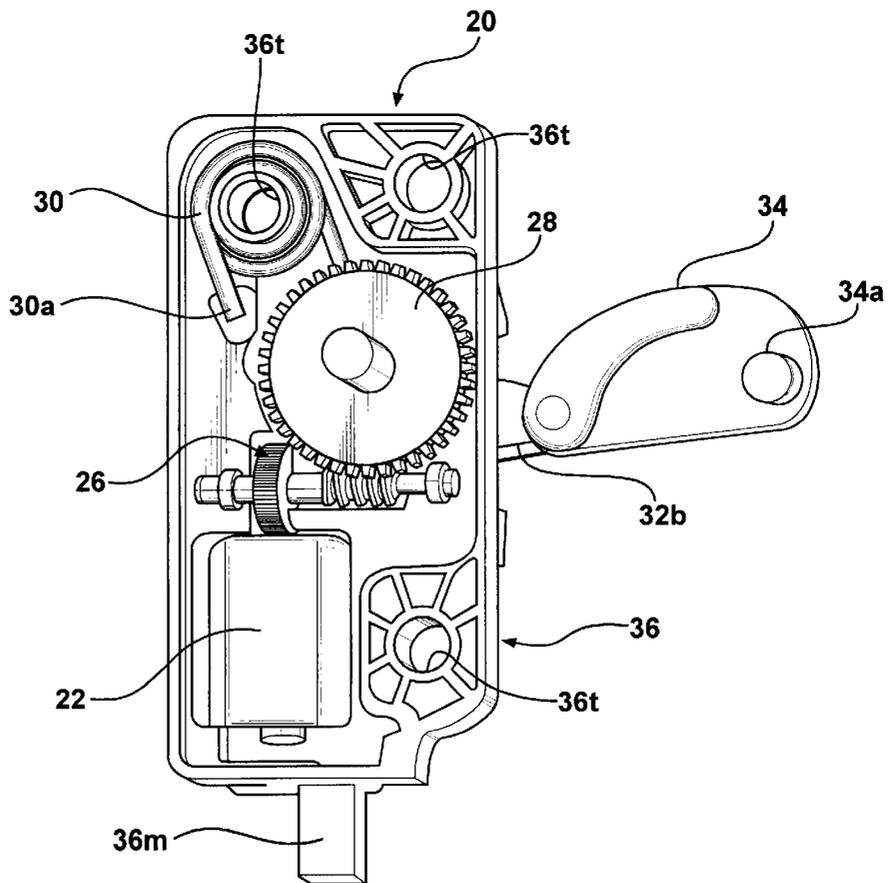


FIG - 3

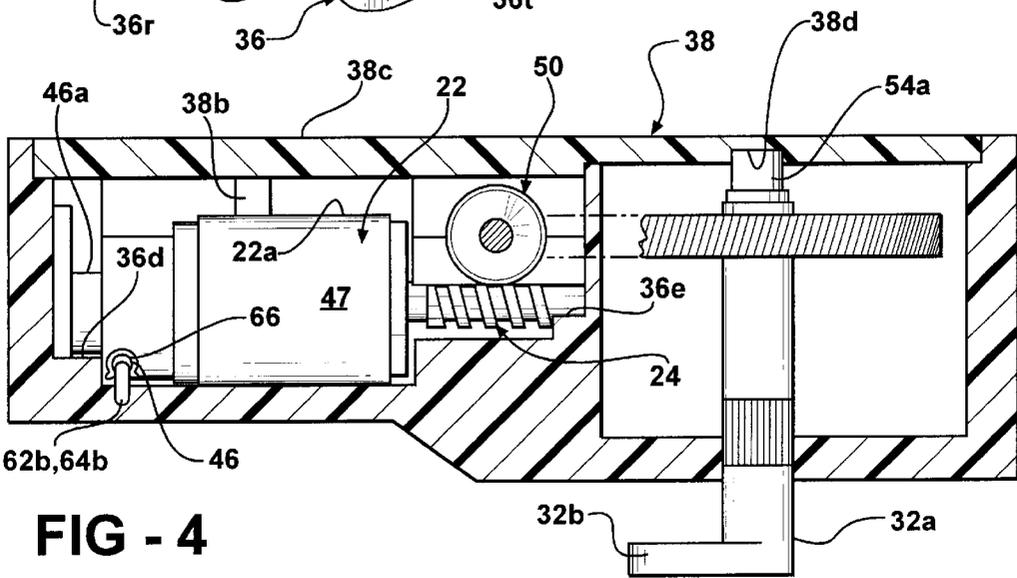
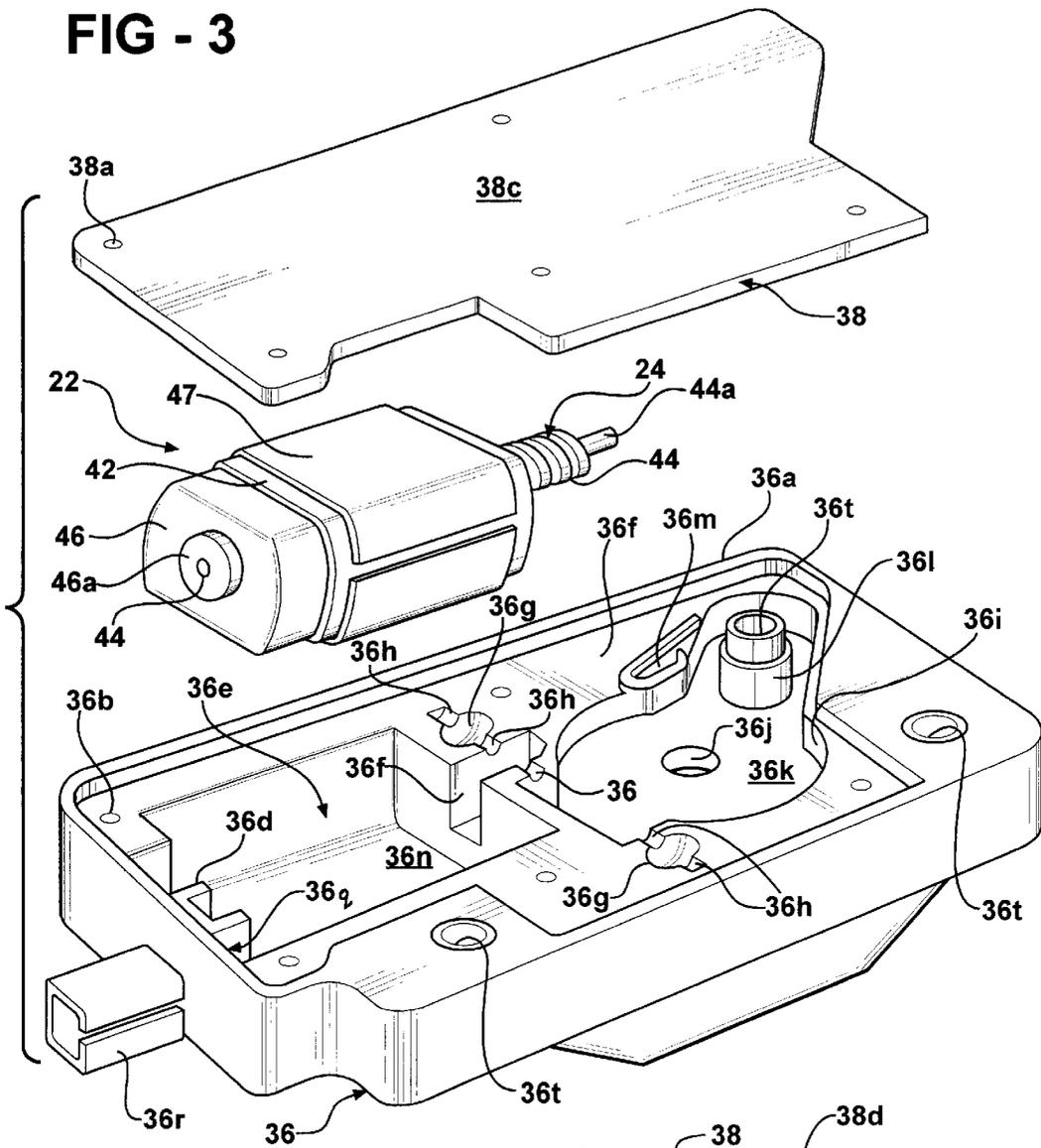


FIG - 4

FIG - 5

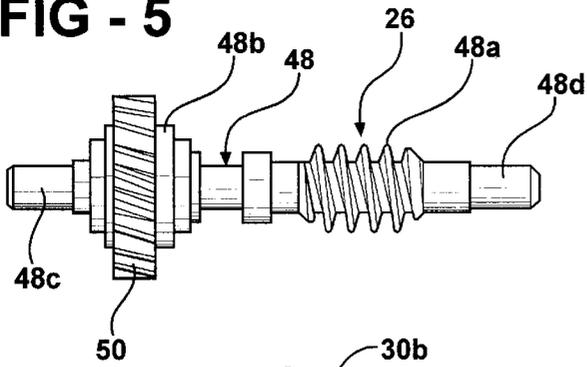


FIG - 6

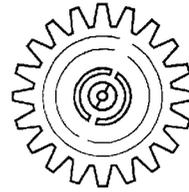


FIG - 7

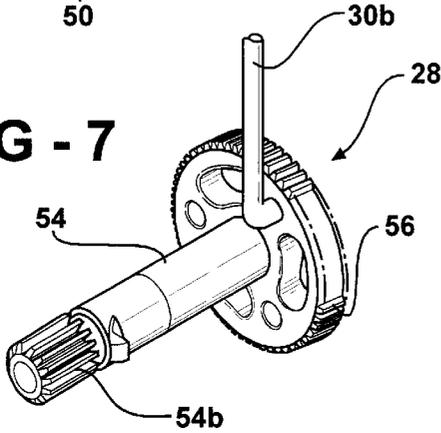


FIG - 8

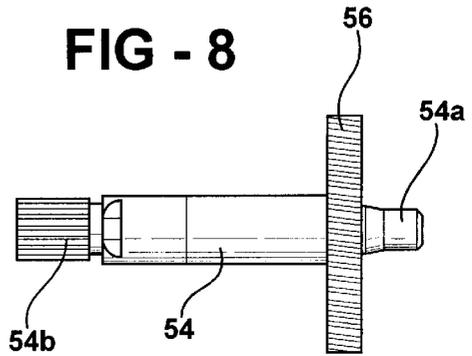


FIG - 9

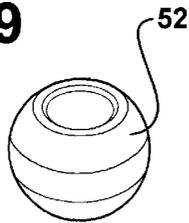


FIG - 10

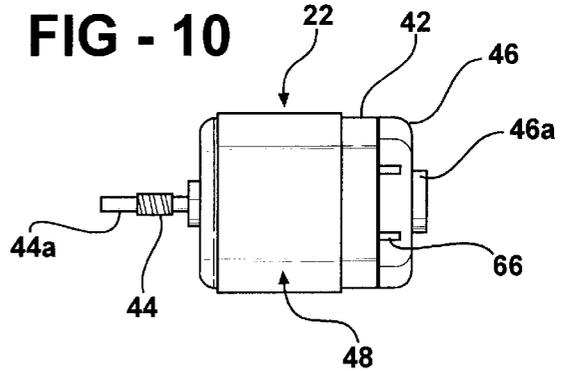


FIG - 11

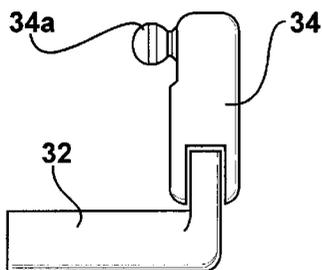


FIG - 12

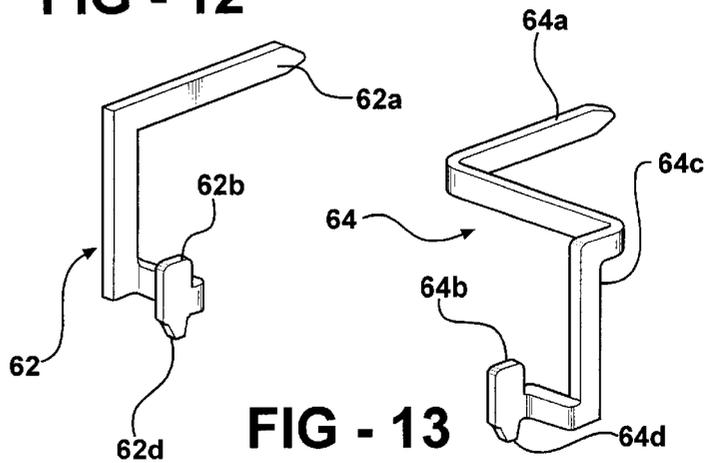


FIG - 13

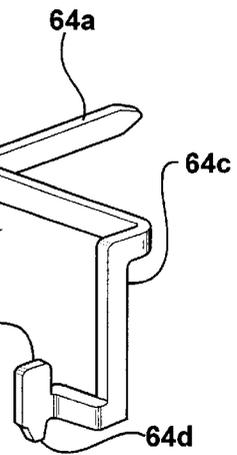


FIG - 14

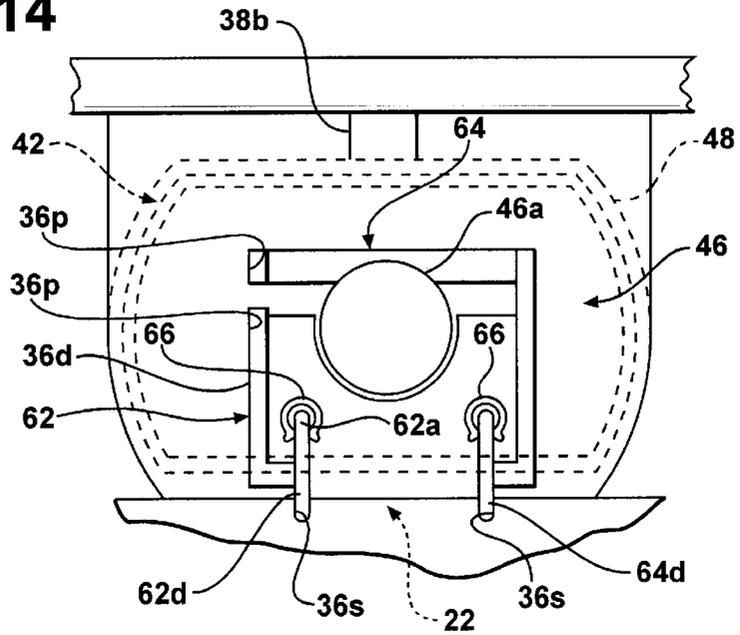


FIG - 15

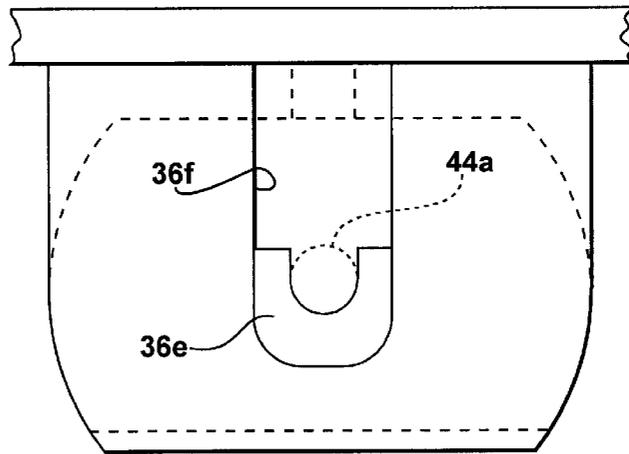
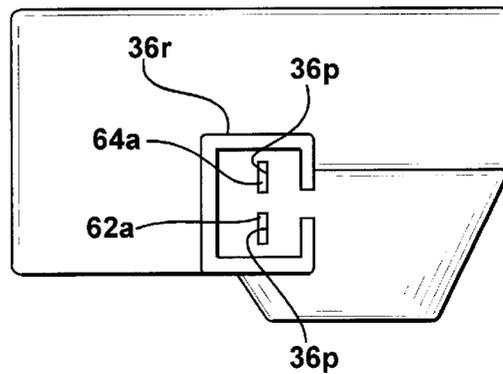


FIG - 16



VENT WINDOW ACTUATOR**FIELD OF THE INVENTION**

This invention relates to a power window actuator and particularly to a power window actuator for a swingably or pivotally mounted window such as a rear side vent or corner window of a motor vehicle such as a van.

BACKGROUND OF THE INVENTION

It is desirable to provide a powered rear side vent window for ventilation purposes in vehicles, particularly vans and minivans. These rear side vent windows are generally swingably mounted and open outwardly in the vehicle body and are typically remotely activated as, for example, from the driver's seat.

Several types of vehicle power vent window actuators are known and used. Whereas these prior art actuators are generally satisfactory, they typically suffer from one or more disadvantages. Specifically, the prior art actuators are relatively expensive and/or overly large and/or are relatively noisy in operation.

SUMMARY OF THE INVENTION

This invention is directed to an improved vent window actuator.

More specifically, this invention is directed to a vent window actuator that is inexpensive, quiet in operation, and compact.

The actuator of the invention is of the type including a housing, an electric motor mounted on the housing, a pivot arm mounted on the housing and adapted to be connected to the vent window to effect opening and closing movement of the vent window in response to pivotal movement of the pivot arm, and a drive mechanism interconnecting the output of the motor and the pivot arm and operative in response to energization of the motor to pivot the pivot arm and move the vent window.

According to the invention, the drive mechanism comprises a first worm driven by the motor, a first worm wheel driven by the first worm, a second worm driven by the first worm wheel, and a second worm wheel driven by the second worm and driving the pivot arm. This arrangement, employing two successive worm drives, provides the quiet operation inherent in a worm drive and, by virtue of the speed reduction achieved utilizing the successive worm drives, allows the use of a smaller, less expensive motor.

According to a further feature of the invention, the first worm wheel is formed of a plastic material. This choice of material, which is made possible by the fact that the back drive loading imposed on the actuator is largely absorbed by the second worm drive, allows the cost of the actuator to be further reduced by virtue of the relatively inexpensive plastic material of the first worm wheel.

According to a further feature of the invention, the actuator further includes a coil spring mounted on the housing proximate the second worm wheel and operative to assist the second worm wheel in generating a sealing force sufficient to positively seal the vent window against the window seal of the motor vehicle. This arrangement allows the actuator to satisfy even very high window sealing requirements irrespective of the use of a relatively small, low power motor.

According to a further feature of the invention, the drive mechanism comprises a first shaft extending coaxial with the

motor output and driven by the motor, a second shaft driven by the first shaft and extending perpendicular to the first shaft, and a third shaft extending perpendicular to the first and second shaft, driven by the second shaft, and driving the pivot arm. These three drive shafts arranged in a mutually perpendicular or cross relationship facilitate the compact packaging of the drive mechanism of the actuator and ensure a robust and smooth transmission of power from the motor to the pivot arm.

According to a further feature of the invention, the motor includes a hub at one end thereof and an output shaft at another end thereof; the first worm is mounted on the output shaft; the housing includes a cavity, a first saddle proximate one end of the cavity, and a second saddle proximate another end of the cavity; and the motor is mounted in the cavity with a free end of the output shaft journaled in the first saddle and the motor hub seated in the second saddle. This arrangement allows a firm, positive mounting of the motor in the housing whereby to eliminate motor wobble and consequent excessive wear in the drive mechanism.

Other applications of the present invention will become apparent to those skilled in the art when the following description of the best mode contemplated for practicing the invention is read in conjunction with the accompanying drawings.

BRIEF DESCRIPTION OF THE DRAWINGS

The description herein makes reference to the accompanying drawings wherein like reference numerals refer to like parts throughout the several views, and wherein:

FIG. 1 is a fragmentary perspective view of a motor vehicle employing a vent window actuator according to the invention;

FIG. 2 is a perspective view of the actuator with a housing cover member removed;

FIG. 3 is a perspective view of the main body member of the actuator housing also showing, in exploded fashion, the actuator motor and the housing cover;

FIG. 4 is a detailed sectional view showing the mounting of the motor in the housing;

FIG. 5 is a side elevational view of an output worm assembly utilized in the actuator;

FIG. 6 is an end view of the output worm assembly;

FIG. 7 is a perspective view of an output shaft assembly utilized in the actuator;

FIG. 8 is a side elevational view of the output shaft assembly;

FIG. 9 is a perspective view of bearing utilized in the actuator;

FIG. 10 is a detailed view of the actuator motor;

FIG. 11 is a view of an actuator arm assembly employed in the actuator;

FIGS. 12 and 13 are perspective views of terminals utilized in the actuator;

FIGS. 14 and 15 are detail views showing the mounting of the motor in the actuator housing; and

FIG. 16 is a detail end view of the actuator housing.

DESCRIPTION OF THE PREFERRED EMBODIMENT

The actuator 10 of the invention is seen in FIG. 1 in association with a motor vehicle 12 which is shown fragmentarily. The motor vehicle 12 may be of a minivan type

and includes a D pillar 14, and a vent window 16 pivotally mounted in a vent window opening 18 for movement by the actuator 10 between open and closed positions.

Actuator 10 (FIGS. 2 and 3) includes a housing 20, a motor 22, a motor worm 24, an output worm assembly 26, an output shaft assembly 28, a spring 30, a link 32, and a link or handle 34.

Housing 20 is suitably formed in a molding operation of a plastic material such, for example, as a glass-filled polyester. Housing 20 includes a main body housing member 36 and a planar cover or lid 38. Main body housing 36 is suitably configured to accommodate the drive mechanism of the actuator, and cover 38 has a planar configuration and is adapted to be seated within a rim 36a of the main body housing to encapsulate the drive mechanism within the housing. The cover may be secured to the main body housing utilizing suitable fasteners coacting with holes 38a in the cover and holes 36b in the main body housing. Housing 20 is secured to D pillar 14 (FIG. 1) utilizing suitable fasteners passing through apertures 36f in main body housing 36.

Motor 22 is sized to fit in a recess or cavity 36c defined by main body housing 36 and comprises a fractional horse power 6000 rpm direct current motor of the type typically used to control automotive door lock circuits. The motor, for example, may comprise a motor available from Johnson Electric Industrial Manufacturing Ltd. of Hong Kong, China as part number NF243G/NS/25028/4E0. Motor 22 includes a main body housing 42, an output shaft 44, a connector 46, and a flux jacket 47. Flux jacket 47 is positioned in surrounding relation to housing 42 and connector 46, which may be formed of a suitable plastic material, is received in an open end of the housing 42 and includes a hub portion 46a journaling the rear end of output shaft 44.

Motor 22 (see also FIGS. 4, 14 and 15) is positioned in cavity 36c with hub 46a seated in a U-shaped saddle 36d defined proximate one end of cavity 36c and with the free end 44a of output shaft 44 journaled in a U-shaped saddle 36e defined at the blind end of a groove 36f opening in the end of cavity 36c opposite saddle 36d. Motor worm 24 may be formed of a suitable brass material and is press fit onto motor output shaft 44 to expose the free end 44a of the output shaft for journaling in saddle 36e. Worm 24 may, for example, have a diametral pitch of 48 teeth per inch and a lead angle of 12° right hand.

Output worm assembly 26 (FIGS. 2, 5, and 6) includes a shaft 48, an output worm 48a formed integrally with the shaft, and a worm wheel 50. Worm wheel 50 is formed of a suitable plastic material, such as nylon, and is press fit on a hub portion 48b of shaft 48 utilizing suitable knurling on the hub portion. Worm wheel 50 has helical teeth with a helix angle of 12° right hand and a diametral pitch of 48 teeth per inch.

Shaft 48 and worm 48a are formed of a suitable ferrous material, such, for example, has SAE steel 12 L14. Output worm 48a has a lead angle of six° right hand and a diametral pitch of 48 teeth per inch.

Output worm assembly 26 is positioned in housing main body 36 utilizing spherical bearings 52 (FIG. 9) received with a press fit in spherical sockets 36g (FIG. 3) opening at laterally-spaced locations in the upper face 36f of housing 36. A first spherical bearing 52 receives one free end 48c of shaft 48 and a second spherical bearing 52 receives the other free end 48d of the shaft to mount the shaft axis at right angles with respect to the axis of the output shaft of the motor with worm wheel 50 positioned in the groove 36f in

meshing engagement with motor worm 24 and with suitable saddle cutouts 36h in the housing accommodating the various intermediate segments of the shaft 48.

Output shaft assembly 28 (FIGS. 2, 7, and 8) comprises an output shaft 54 and an output gear 56 both formed of a suitable ferrous material. Gear 56 is press fit on one end of shaft 54 to expose a pilot or journal portion 54a and includes straight-cut gears having a diametral pitch of 48 teeth per inch.

Output shaft assembly 28 is positioned in housing member 36 with gear 56 positioned in a circular cavity 36i opening in the upper face 36f of the housing and arranged in meshing engagement with output worm 48a with the portion of shaft 54 remote from pilot end 54a extending downwardly in a perpendicular journal bore 36j opening in a recessed face 36k of the housing. The axis of shaft 54 is thus positioned perpendicular or crossed with respect to the axis of shaft 48 and perpendicular or crossed with respect to the axis of the motor output shaft 44.

Spring 30 comprises a coil spring 30 positioned in surrounding relation to a hollow post 36l upstanding from housing face 36k in surrounding relation to an aperture 36f with one end 30a of the spring anchored in an anchor socket 36m defined by the housing and the other end 30b (FIG. 7) of the spring hooked into a hole in output gear 56. The spring is arranged to assist movement of output gear 56 and output shaft 54 in a direction to close vent window 16.

With cover 38 in place over housing member 36, a finger 38b (FIGS. 4 and 14) downstanding from the main planar body 38c of the cover engages the upper face 22a of motor 22 to press connector hub portion 46a downwardly in saddle 36d and press motor output shaft 44a downwardly in saddle 36e and a socket 38d defined on the lower face of the cover journals the upper pilot end 44a of shaft 54. Socket 38d is coaxial with the axis of bore 36j whereby to provide precise perpendicular or cross positioning of the axis of shaft 54 relative to the axis of shaft 48 and relative to the axis of the motor output shaft. The in place cover also serves to trap bearings 52 in sockets 36j whereby to firmly position the axis of shaft 48 and further serves to preclude upward displacement of spring 30. The dimensional parameters are chosen such that (1) the lower face 22b of the in place motor is spaced above the floor 36n of cavity 36c so that the in place motor is supported solely by saddles 36d and 36e and finger 38b, and (2) finger 38b is compressed between the main body of the cover and the upper face of the motor to pressure seat hub portion 46a and output shaft 44a in the respective saddles.

Link 32 (FIGS. 1, 2, and 11) includes a main body portion 32a and a crank or pivot arm portion 32b. Main body portion 32a is internally splined and is fitted over external splines 54b on shaft 54 within bore 36j of the housing and extends outwardly from the lower face of the housing to position pivot arm portion 32b exteriorly of the housing.

Handle 34 is suitably pivotally secured to the free end of pivot arm portion 32b and includes a ball socket joint 34a at its free end for suitable connection to a bracket 60 suitably secured to an inner face of vent window 16.

Power is provided to the motor 22 via a pair of terminals 62 and 64. Each terminal defines a prong or plug portion 62a/64a, a contact portion 62b/64b, and a bridge portion 62c/64c interconnecting the prong and contact portions.

When assembling the actuator, terminal prong portions 62a, 64a are passed through apertures 36p (FIGS. 14 and 16) in housing end wall 36q proximate saddle 36d to position the prong portions 62a/64a in on edge stacked relation in an

external socket housing **36r** formed integrally with housing member **36** and extending from housing wall **36q** and position pilot portions **62d/64d** of contacts **62b/64b** in laterally spaced locator holes **36s** in housing floor **36n** proximate saddle **36d**. Motor **22** is now installed in cavity **36c** by positioning connector hub **46a** in saddle **36d** and positioning output shaft **44a** in saddle **36e** while simultaneously inserting contacts **62b/64b** in laterally spaced female sockets **66** (FIGS. **4** and **14**) mounted in the lower face of connector **46**, whereby to power the actuator simply by inserting an electrical plug into the open end of socket housing **36r** to access the stacked prongs **62a/64a**.

The ratios of the successive worm drive assembly are chosen such as to provide an overall drive ratio between the output shaft of the motor and the link **32** of 600:1 and the parameters of the vent window are chosen such that the window undergoes a movement of 135° between open and shut positions.

The vent window actuator of the invention will be seen to provide many important advantages. Specifically, the overall drive ratio provided by the successive worm drives allows the use of a relatively small and relatively inexpensive motor thereby providing cost savings. Further cost savings are provided by the fact that the motor may comprise a volume-produced motor already in volume use to control automotive door lock circuits. The cost of the actuator is further reduced by the use of a plastic worm wheel in engagement with the motor worm. The use of a plastic worm wheel engaging the motor worm is made possible by the fact that the worm drive of the output shaft assembly absorbs much of the back loading in the system so that the loading experienced at the interface of the motor worm and the worm wheel of the output worm assembly is sufficiently reduced to allow the use of a non-ferrous, relatively inexpensive material at this interface. The use of two successive worm drives also provides quieter operation for the actuator as compared to prior art actuators employing metal-to-metal drive throughout and the cross axis gearing defined by the successive worm drives provides a compact package allowing a reduction in the overall size of the actuator. The actuator package size is further reduced by the use of a smaller motor. The cross axis arrangement of the gearing also reduces shaft misalignment problems and provides a smooth, robust drive mechanism. The mounting of the motor in the housing utilizing the saddles engaging the free end of the motor output shaft and the hub portion on the integral connector in combination with the finger of the cover provides firm mounting for the motor whereby to reduce motor wobble and minimize wear in the drive system occasioned by misalignments. The use of a connector formed integrally with the motor and positioned within the housing of the actuator further reduces the complexity and cost of the unit since the connector in known manner incorporates built-in RFI provisions so that there is no need to separately provide radio frequency interference protection for the actuator. The use of the spherical bearings mounting the output shaft assembly provides a further self-aligning feature whereby to further minimize wear in the system due to misalignments in the drive assembly. The use of the separate assist spring coating with the output shaft assembly allows the use of a relatively small motor even in installations requiring a high sealing force as between the vent window and the adjacent window seal of the motor vehicle. The integration of the connector into the motor and thereby into the housing combined with the terminals extending from the outside of the housing provide a simple and inexpensive electrical connection for the motor, and the rigid enclosed box design

of the housing adds strength to the actuator as compared to prior art actuators utilizing a relatively weak open section housing.

While the invention has been described in connection with what is presently considered to be the most practical and preferred embodiment, it is to be understood that the invention is not to be limited to the disclosed embodiments but, on the contrary, is intended to cover various modifications and equivalent arrangements included within the spirit and scope of the appended claims, which scope is to be accorded the broadest interpretation so as to encompass all such modifications and equivalent structures as is permitted under the law.

What is claimed is:

1. An actuator for a vent window of a motor vehicle, the actuator including a housing, an electric motor mounted on the housing, and including an output shaft, a pivot arm mounted on the housing and adapted to be connected to the vent window to effect opening and closing movement of the vent window in response to pivotal movement of the pivot arm, and a drive mechanism interconnecting the output of the motor and the pivot arm and operative in response to energization of the motor to pivot the pivot arm and move the vent window, characterized in that:

the drive mechanism comprises a first worm driven by the motor output shaft, a first worm wheel driven by the first worm, a second worm driven by the first worm wheel, and a second worm wheel driven by the second worm and driving the pivot arm; and

the actuator further includes a coil spring mounted on the housing proximate the second worm wheel and operative to assist the second worm wheel in generating a sealing force sufficient to positively seal the vent window against a window seal of the motor vehicle.

2. An actuator for a vent window of a motor vehicle, the actuator including a housing, an electric motor mounted on the housing, a pivot arm mounted on the housing and adapted to be connected to the vent window to effect opening and closing movement of the vent window in response to pivotal movement of the pivot arm, and a drive mechanism interconnecting the output of the motor and the pivot arm and operative in response to energization of the motor to pivot the pivot arm and move the vent window, characterized in that:

the drive mechanism comprises a first shaft extending coaxially forwardly from the motor output shaft and driven by the motor, a second shaft driven by the first shaft and extending perpendicular to the first shaft, and a third shaft positioned forwardly of the second shaft, extending perpendicular to the first and second shafts, driven by the second shaft, and driving the pivot arm.

3. An actuator according to claim **2** wherein:

the drive mechanism further includes a first worm driven by the first shaft;

the second shaft is driven by a first worm wheel driven by the first worm; and

the third shaft is driven by a second worm wheel driven by the second worm.

4. An actuator according to claim **3** wherein the first worm wheel is formed of a plastic material.

5. An actuator according to claim **3** wherein:

the motor includes a hub at one end thereof and an output shaft at another end thereof comprising the first shaft;

the first worm is mounted on the motor output shaft;

the housing includes a cavity, a first saddle proximate one end of the cavity, and a second saddle proximate another end of the cavity; and

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the motor is mounted in the cavity with a free end of the output shaft journaled in the first saddle and the motor hub seated in the second saddle.

6. An actuator according to claim 2 wherein:

the housing includes a main body housing member and a generally planar housing cover;

the main body housing member defines a cavity receiving the motor and further defines a bore journaling the third shaft; and

the cover defines a socket coaxial with the bore and journaling a free end of the third shaft.

7. An actuator for a vent window of a motor vehicle, the actuator including a housing, an electric motor mounted on the housing, a pivot arm mounted on the housing and adapted to be connected to the vent window to effect opening and closing movement of the vent window in response to pivotal movement of the pivot arm, and a drive mechanism interconnecting the output of the motor and the pivot arm and operative in response to energization of the motor to pivot the pivot arm and move the vent window, characterized in that:

the drive mechanism comprises a first shaft extending coaxial with the motor output and driven by the motor, a second shaft driven by the first shaft and extending perpendicular to the first shaft, and a third shaft extending perpendicular to the first and second shafts, driven by the second shaft, and driving the pivot arm;

the drive mechanism further includes a first worm driven by the first shaft;

the second shaft is driven by a first worm wheel driven by the first worm;

the third shaft is driven by a second worm wheel driven by the second worm; and

the actuator further includes a coil spring mounted on the housing proximate the second worm wheel and operative to assist the second worm wheel in generating a force sufficient to positively seal the vent window against a window seal of the motor vehicle.

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8. An actuator for a vent window of a motor vehicle, the actuator including a housing, an electric motor mounted on the housing, a pivot arm mounted on the housing and adapted to be connected to the vent window to effect opening and closing movement of the vent window in response to pivotal movement of the pivot arm, and a drive mechanism interconnecting the output of the motor and the pivot arm and operative in response to energization of the motor to pivot the pivot arm and move the vent window, characterized in that:

the drive mechanism comprises a first shaft extending coaxial with the motor output and driven by the motor, a second shaft driven by the first shaft and extending perpendicular to the first shaft, and a third shaft extending perpendicular to the first and second shafts, driven by the second shaft, and driving the pivot arm;

the drive mechanism further includes a first worm driven by the first shaft;

the second shaft is driven by a first worm wheel driven by the first worm;

the third shaft is driven by a second worm wheel driven by the second worm;

the motor includes a hub at one end thereof and an output shaft at another end thereof comprising the first shaft;

the first worm is mounted on the motor output shaft;

the housing includes a cavity, a first saddle proximate one end of the cavity, and a second saddle proximate another end of the cavity;

the motor is mounted in the cavity with a free end of the output shaft journaled in the first saddle and the motor hub seated in the second saddle;

the housing includes a main body member defining the cavity and the saddles and a generally planer cover closing the cavity; and

the cover includes a finger extending downwardly to engage an upper face of the motor to positively load the output shaft and the hub into the respective saddles.

* * * * *