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DESCRIPTION

[0001] The invention is related to an ejector refrigeration circuit, in particular to an ejector refrigeration circuit comprising at least two controllable ejectors and a method of controlling said ejectors.

[0002] Controllable ejectors may be used in refrigeration circuits as high pressure control devices for controlling the high pressure level of a circulating refrigerant by varying the high pressure mass flow of the refrigerant rate through the ejector. The variable high pressure mass flow is controllable by the ejector opening degree and can be adjusted between zero and one hundred percent. An ejector additionally may operate as a so called ejector pump for compressing refrigerant from a low pressure level to a medium pressure level using energy that becomes available when expanding the refrigerant from a high pressure level to the medium pressure level.

[0003] JP 2010 151424 A discloses an air conditioning device including a refrigerant circuit constituted by connecting a compressor, a condenser, ejectors, a gas-liquid separator for the ejectors, and an evaporator and performing a refrigerating cycle. The plurality of ejectors are disposed in parallel with each other. The plurality of ejectors include a variable opening ejector which can regulate a flow rate by controlling an opening of a throat section of a rubber nozzle by a needle valve, and a fixed opening ejector which cannot regulate the flow rate.

[0004] US 2013 111 935 A1 discloses a system with a compressor, a heat rejection heat exchanger, first and second ejectors, first and second heat absorption heat exchangers, and a separator. The ejectors each have a primary inlet coupled to the heat rejection exchanger to receive refrigerant. A second heat absorption heat exchanger is coupled to the outlet of the second ejector to receive refrigerant. The separator has an inlet coupled to the outlet of the first ejector to receive refrigerant from the first ejector. The separator has a gas outlet coupled to the secondary inlet of the second ejector to deliver refrigerant to the second ejector. The separator has a liquid outlet coupled to the secondary inlet of the first ejector via the first heat absorption heat exchanger to deliver refrigerant to the first ejector. The ejector may be controllable ejectors. The controllable ejectors may be used to control the high-side pressure and the capacities of each evaporator. To increase the capacity of the first evaporator, the first ejector may be opened. To increase the capacity of the second evaporator, the second ejector may be opened.

[0005] It would be beneficial to optimize the efficiency of an ejector refrigeration circuit for any given overall high pressure mass flow.

[0006] Exemplary embodiments of the invention include a method of operating an ejector refrigeration circuit with at least two controllable ejectors connected in parallel and respectively comprising a controllable motive nozzle, a controllable primary high pressure input port forming an inlet to the controllable motive nozzle, a secondary low pressure input port and a medium

pressure output port, wherein the method comprises the steps of:

1. a) operating a first ejector of the at least two controllable ejectors by controlling the opening degree of its controllable primary high pressure input port until the maximum efficiency of said first ejector has been reached or the actual refrigeration demands are met;
2. b) operating at least one additional ejector of the at least two controllable ejectors by gradually opening its controllable primary high pressure input port for increasing the refrigeration capacity of the ejector refrigeration circuit in case the actual refrigeration demands are not met by operating the first ejector alone.

[0007] Exemplary embodiments of the invention also include an ejector refrigeration circuit, which is configured for circulating a refrigerant, in particular carbon dioxide, and comprises:

at least two controllable ejectors connected in parallel and respectively comprising a controllable motive nozzle, a controllable primary high pressure input port forming an inlet to the controllable motive nozzle, a secondary low pressure input port and a medium pressure output port; and

a control unit, which is configured for operating the ejector refrigeration circuit employing a method comprising the steps of:

1. a) operating a first ejector of the at least two controllable ejectors by controlling the opening degree of its controllable high pressure port until the maximum efficiency of said first ejector has been reached or the actual refrigeration demands are met;
2. b) operating at least one additional ejector of the at least two controllable ejectors by gradually opening its controllable primary high pressure input port for increasing the refrigeration capacity of the ejector refrigeration circuit in case the actual refrigeration demands are not met by operating the first ejector alone.

[0008] The efficiency of an individual ejector is a function of the high pressure mass flow rate while the overall high pressure mass flow, i.e. the mass flow through all ejectors, is given as a control input via the needed high pressure drop. In order to cope with part load operation an ejector refrigeration circuit according to exemplary embodiments of the invention is equipped with at least two controllable ejectors, which are configured for working in parallel.

[0009] Operating an ejector refrigeration circuit comprising at least two controllable ejectors according to exemplary embodiments of the invention allows to operate the ejector refrigeration circuit very stable and efficiently, as it reliably avoids to operate any of the controllable ejectors in a range of operation in which its operation is less efficient. This results in an optimized efficiency of the ejector refrigeration circuit over a wide range of operational conditions.

Short Description of the Figures:**[0010]**

An exemplary embodiment of the invention will be described in the following with respect to the enclosed Figures.

Figure 1 illustrates a schematic view of an ejector refrigeration circuit according to an exemplary embodiment of the invention.

Figure 2 illustrates a schematic sectional view of a controllable ejector as it may be employed in the exemplary embodiment shown in Figure 1.

Detailed Description of the Figures:

[0011] Figure 1 illustrates a schematic view of an ejector refrigeration circuit 1 according to an exemplary embodiment of the invention comprising a high pressure ejector circuit 3, a refrigerating evaporator flowpath 5, and a low temperature flowpath 9 respectively circulating a refrigerant as indicated by the arrows F_1 , F_2 , and F_3 .

[0012] The high pressure ejector circuit 3 comprises a compressor unit 2 including a plurality of compressors 2a, 2b, 2c connected in parallel.

[0013] The high pressure side outlets 22a, 22b, 22c of said compressors 2a, 2b, 2c are fluidly connected to an outlet manifold collecting the refrigerant from the compressors 2a, 2b, 2c and delivering the refrigerant via a heat rejection heat exchanger/gas cooler inlet line to the inlet side 4a of a heat rejecting heat exchanger/gas cooler 4. The heat rejecting heat exchanger/gas cooler 4 is configured for transferring heat from the refrigerant to the environment for reducing the temperature of the refrigerant. In the exemplary embodiment shown in Figure 1, the heat rejecting heat exchanger/gas cooler 4 comprises two fans 38 which are operable for blowing air through the heat rejecting heat exchanger/gas cooler 4 in order to enhance the transfer of heat from the refrigerant to the environment. Of course, the fans 38 are optional and their number may be adjusted to the actual needs.

[0014] The cooled refrigerant leaving the heat rejecting heat exchanger/gas cooler 4 at its outlet side 4b is delivered via a high pressure input line 31 comprising a service valve 20 to primary high pressure inlet ports 6a, 7a of two controllable ejectors 6, 7, which are connected in parallel and configured for expanding the refrigerant to a reduced pressure level. The service valve 20 allows to shut down the flow of refrigerant to the primary high pressure input ports 6a, 7a in case an ejector 6, 7 needs to be maintained or replaced.

[0015] Details of the controllable ejectors 6, 7 will be described further below with reference to Fig. 2.

[0016] The expanded refrigerant leaves the controllable ejectors 6, 7 through respective ejector output ports 6c, 7c and is delivered by means of an ejector output line 35 to an inlet 8a of a receiver 8. Within the receiver 8, the refrigerant is separated by gravity into a liquid portion collecting at the bottom of the receiver 8 and a gas phase portion collecting in an upper part of the receiver 8.

[0017] The gas phase portion of the refrigerant leaves the receiver 8 through a receiver gas outlet 8b provided at the top of the receiver 8. Said gas phase portion is delivered via a receiver gas outlet line 40 to the inlet sides 21a, 22b, 22c of the compressors 2a, 2b, 2c, which completes the refrigerant cycle of the high pressure ejector circuit 3.

[0018] Refrigerant from the liquid phase portion of the refrigerant collecting at the bottom of the receiver 8 exits from the receiver 8 via a liquid outlet 8c provided at the bottom of the receiver 8 and is delivered through a receiver liquid outlet line 36 to the inlet side 10a of a refrigeration expansion device 10 ("medium temperature expansion device") and, optionally, to a low temperature expansion device 14.

[0019] After having left the refrigeration expansion device 10, where it has been expanded, via its outlet side 10b, the refrigerant enters into a refrigeration evaporator 12 ("medium temperature evaporator"), which is configured for operating at "normal" cooling temperatures, in particular in a temperature range of -10 °C to +5 °C, for providing medium temperature refrigeration.

[0020] After having left the refrigeration evaporator 12 via its outlet 12b, the refrigerant flows through a low pressure inlet line 33 to the inlet sides of two ejector inlet valves 26, 27. The outlet sides of said ejector inlet valves 26, 27, which preferably are provided as non-adjustable shut-off valves, are respectively connected to the secondary low pressure inlet ports 6b, 7b of the controllable ejectors 6, 7. In case the respective ejector inlet valve 26, 27 is open, the refrigerant leaving the refrigeration evaporator 12 is sucked into the associated controllable ejector 6, 7 by means of the high pressure flow entering via the respective ejector's 6, 7 primary high pressure inlet port 6a, 7a. This functionality of the controllable ejectors 6, 7 providing an ejector pump will be described in more detail below with reference to Figure 2.

[0021] The portion of the liquid refrigerant that has been delivered to and expanded by the optional low temperature expansion device 14 enters into an optional low temperature evaporator 16, which in particular is configured for operating at low temperatures in particular at temperatures in the range of -40 °C to -25 °C. After having left the low temperature evaporator 16 the refrigerant is delivered to the inlet side of a low temperature compressor unit 18 comprising one or more, in the embodiment shown in Figure 1 two, low temperature compressors 18a, 18b.

[0022] In operation, the low temperature compressor unit 18 compresses the refrigerant supplied by the low temperature evaporator 16 to medium pressure, i.e. basically the same pressure as the pressure of the refrigerant which is delivered from the gas outlet 8b of the receiver 8. The compressed refrigerant is supplied together with the refrigerant provided from the gas outlet 8b of the receiver 8 to the inlet sides 21a, 21b, 21c of the compressors 2a, 2b, 2c.

[0023] Sensors 30, 32, 34 which are configured for measuring the pressure and/or the temperature of the refrigerant are respectively provided at the high pressure input line 31 fluidly connected to the primary high pressure input ports 6a, 7a of the controllable ejectors 6, 7, the low pressure input line 33 fluidly connected to the secondary low pressure input ports 6b, 7b and the output line 35 fluidly connected to the ejector output ports 6c, 7c. A control unit 28 is configured for controlling the operation of the ejector refrigeration circuit 1, in particular the operation of the compressors 2a, 2b, 2c, 18a, 18b, the controllable ejectors 6, 7 and the controllable valves 26, 27 provided at the secondary low pressure input ports 6b, 7b of the controllable ejectors 6, 7 based on the pressure value(s) and/or the temperature value(s) provided by the sensors 30, 32, 34 and the actual refrigeration demands.

[0024] In a first mode of operation, when the refrigeration demands and/or the ambient temperature at the heat rejecting heat exchanger/gas cooler 4 are relatively low, only a single (first) ejector 6 of the controllable ejectors 6, 7 is operated, while both, the primary high pressure inlet port 7a and the low pressure inlet valve 27 of the second ejector 7 are closed. With increasing refrigeration demands and/or increasing ambient temperatures at the heat rejecting heat exchanger/gas cooler 4 the primary high pressure inlet port 6a of the first controllable ejector 6 is gradually opened until the actual refrigeration demands are met or the optimal point of operation of the first controllable ejector 6 is reached. In case the optimal point of operation of the first controllable ejector 6 is reached before the actual refrigeration demands are met, the primary high pressure inlet port 7a of the second controllable ejector 7 is additionally opened for increasing the refrigeration capacity of the ejector refrigeration circuit 1 in order to meet the increased refrigeration demands without operating the first controllable ejector 6 beyond its optimal point of operation.

[0025] Even when the primary high pressure inlet port 7a of the second controllable ejector 7 is opened, the associated low pressure inlet valve 27 may remain closed for operating the second controllable ejector 7 as a high pressure bypass valve bypassing the first controllable ejector 6. When the opening degree of the primary high pressure inlet port 7a has reached a point allowing the second controllable ejector 7 to run stable and efficiently, the low pressure inlet valve 27 of said second controllable ejector 7 may be opened for increasing the flow of refrigerant flowing through the refrigeration expansion device 10 and the refrigeration evaporator 12.

[0026] Although only two controllable ejectors 6, 7 are shown in Figure 1, it is self-evident that the invention may be applied similarly to ejector refrigeration circuits comprising three or more

controllable ejectors 6, 7 connected in parallel. The controllable ejectors 6, 7 may have the same capacity or different capacities. In particular, the capacity of the second ejector 7 may be twice as large as the capacity of the first ejector 6, the capacity of an optional third ejector (not shown) may be twice as large as the capacity of the second ejector 7 etc. Such an ejector configuration provides a wide range of available capacities by allowing to selectively operate a suitable combination of controllable ejectors 6, 7.

[0027] In case a plurality of controllable ejectors 6, 7 having the same capacity are provided, every ejector 6, 7 alternately may be used as the first ejector 6, i.e. as the ejector 6 operated alone at low refrigeration demands and/or low ambient temperatures. This will result in a uniform wear of the controllable ejectors 6, 7 reducing the costs for maintenance.

[0028] In case the controllable ejectors 6, 7 are provided with different capacities, any from the plurality of controllable ejectors 6, 7 may be selected to operate alone acting as the "first ejector" based on the actual refrigeration demands and/or ambient temperatures in order to enhance the efficiency of the ejector refrigeration circuit by using the controllable ejector 6, 7 which may be operated closest to its optimal point of operation.

[0029] Figure 2 illustrates a schematic sectional view of an exemplary embodiment of a controllable ejector 6 as it may be employed as each of the controllable ejectors 6, 7 in the ejector refrigeration circuit 1 shown in Figure 1.

[0030] The ejector 6 is formed by a motive nozzle 100 nested within an outer member 102. The primary high pressure inlet port 6a forms the inlet to the motive nozzle 100. The ejector output port 6c is the outlet of the outer member 102. A primary refrigerant flow 103 enters via the primary high pressure inlet port 6a and then passes into a convergent section 104 of the motive nozzle 100. It then passes through a throat section 106 and a divergent expansion section 108 to an outlet 110 of the motive nozzle 100. The motive nozzle 100 accelerates the flow 103 and decreases the pressure of the flow. The secondary low pressure inlet port 6b forms an inlet of the outer member 102. The pressure reduction caused to the primary flow by the motive nozzle draws a secondary flow 112 from the secondary low pressure inlet port 6b into the outer member 102. The outer member 102 includes a mixer having a convergent section 114 and an elongate throat or mixing section 116. The outer member 102 also has a divergent section ("diffuser") 118 downstream of the elongate throat or mixing section 116. The motive nozzle outlet 110 is positioned within the convergent section 114. As the flow 103 exits the outlet 110, it begins to mix with the secondary flow 112 with further mixing occurring through the mixing section 116 providing a mixing zone. Thus, respective primary and secondary flowpaths respectively extend from the primary high pressure inlet port 6a and the secondary low pressure inlet port 6b to the ejector output port 6c, merging at the exit.

[0031] In operation, the primary flow 103 may be supercritical upon entering the ejector 6 and subcritical upon exiting the motive nozzle 100. The secondary flow 112 may be gaseous or a mixture of gas comprising a smaller amount of liquid upon entering the secondary low pressure inlet port 6b. The resulting combined flow 120 is a liquid/vapor mixture and decelerates and

recovers pressure in the diffuser 118 while remaining a mixture.

[0032] The exemplary ejectors 6, 7 employed in exemplary embodiments of the invention are controllable ejectors. Their controllability is provided by a needle valve 130 having a needle 132 and an actuator 134. The actuator 134 is configured for shifting a tip portion 136 of the needle 132 into and out of the throat section 106 of the motive nozzle 100 for modulating the flow through the motive nozzle 100 and, in turn, the ejector 6 overall. Exemplary actuators 134 are electric, e.g. solenoid or the like. The actuator 134 is coupled to and controlled by the control unit 28. The control unit 28 may be coupled to the actuator 134 and other controllable system components via hardwired or wireless communication paths. The control unit 28 may include one or more of: processors; memory (e.g., for storing program information for execution by the processor to perform the operational methods and for storing data used or generated by the program(s)); and hardware interface devices (e.g., ports) for interfacing with input/output devices and controllable system components.

Further embodiments:

[0033] A number of optional features are set out in the following. These features may be realized in particular embodiments, alone or in combination with any of the other features.

[0034] In an embodiment the method includes gradually opening the primary high pressure input port of at least one additional controllable ejector in order to adjust the mass flow through the additional controllable ejector to the actual refrigeration demands. Gradually opening the primary high pressure input port allows for an exact adjustment of the mass flow through the additional controllable ejector.

[0035] In an embodiment the method further includes operating at least one of the controllable ejectors with its secondary low pressure input port being closed. A controllable valve, which is preferably provided in the form of a non-adjustable shut-off valve, may be provided upstream the secondary low pressure input port of at least one/each of the controllable ejectors. Such a controllable valve allows to close the respective ejector's secondary low pressure input port for running at least of the controllable ejectors as a bypass high pressure control valve increasing the mass flow of the refrigerant through the heat rejecting heat exchanger/gas cooler in case said ejector would not run stable and efficient with its secondary low pressure input port being open.

[0036] In an embodiment the method further includes opening the secondary low pressure input port of the at least one ejector, which has been operated with its secondary low pressure input port being closed, for increasing the mass flow of the refrigerant through the heat rejecting heat exchanger(s) to meet the actual refrigeration demands.

[0037] In an embodiment the method further includes the step of closing the primary high pressure input port and/or the secondary low pressure input port of the first ejector in case the

ejector refrigeration circuit is operated more efficiently by running only at least one of the additional controllable ejectors.

[0038] In an embodiment the method further includes using carbon dioxide as refrigerant, which provides an efficient and safe, i.e. non-toxic, refrigerant.

[0039] In an embodiment the ejector refrigeration circuit further comprises:

a heat rejecting heat exchanger/gas cooler having an inlet side and an outlet side, wherein the outlet side of the heat rejecting heat exchanger/gas cooler is fluidly connected to the primary high pressure input ports of the controllable ejectors;

a receiver, having a liquid outlet, a gas outlet and an inlet, which is fluidly connected to the outlet ports of the controllable ejectors;

at least one compressor having an inlet side and an outlet side, the inlet side of the at least one compressor being fluidly connected to the gas outlet of the receiver, and the outlet side of the at least one compressor being fluidly connected to the inlet side of the heat rejecting heat exchanger/gas cooler;

at least one refrigeration expansion device having an input side, which is fluidly connected to the liquid outlet of the receiver, and an outlet side; and

at least one refrigeration evaporator, which is fluidly connected between the outlet side of the at least one refrigeration expansion device and the secondary low pressure input ports of the controllable ejectors.

[0040] In an embodiment all of the controllable ejectors are provided with the same capacity. This allows to freely choose between the controllable ejectors and in particular allows to distribute the time of operation equally between the controllable ejectors for causing an even wear of the controllable ejectors.

[0041] In an alternative embodiment the controllable ejectors are provided with different capacities allowing to cover a wide range of operational conditions by operating a selected combination of the controllable ejectors. The controllable ejectors in particular may be provided with doubled capacity ratios, i.e. 1:2:4:8..., in order to cover a wide range of possible capacities.

[0042] In an embodiment at least one sensor, which is configured for measuring the pressure and/or the temperature of the refrigerant, is provided in at least one of a high pressure input line fluidly connected to the primary high pressure input ports, a low pressure input line fluidly connected to the secondary low pressure input ports and an output line fluidly connected to the output ports of the controllable ejectors, respectively. Such sensors allow to optimize the operation of the controllable ejectors based on the pressure value(s) and/or temperature

value(s) provided by the sensor(s).

[0043] In an embodiment at least one service valve is provided upstream of the controllable ejectors' primary high pressure input ports for allowing to shut down the flow of refrigerant to the primary high pressure input ports in case an ejector needs to be maintained or replaced.

[0044] In an embodiment the ejector refrigeration circuit further comprises at least one low temperature circuit which is configured for providing low cooling temperatures in addition to the medium cooling temperatures provided by the refrigerating evaporator flowpath. The low temperature circuit is connected between the liquid outlet of the receiver and the inlet side of the at least one compressor and comprises in the direction of flow of the refrigerant: at least one low temperature expansion device, at least one low temperature evaporator, and at least one low temperature compressor.

[0045] While the invention has been described with reference to exemplary embodiments, it will be understood by those skilled in the art that various changes may be made and equivalence may be substitute for elements thereof without departing from the scope of the invention. In particular, modifications may be made to adapt a particular situation or material to the teachings of the invention without departing from the essential scope thereof. Therefore, it is intended that the invention is not limited to the particular embodiments disclosed, but that the invention will include all embodiments falling within the scope of the pending claims.

Reference Numerals

[0046]

- 1
ejector refrigeration circuit
- 2
compressor unit
- 2a, 2b, 2c
compressors
- 3
high pressure ejector circuit
- 4
heat rejecting heat exchanger/gas cooler
- 4a
inlet side of the heat rejecting heat exchanger/gas cooler
- 4b
outlet side of the heat rejecting heat exchanger/gas cooler
- 5
refrigerating evaporator flowpath
- 6

- first controllable ejector
- 6a primary high pressure inlet port of the first controllable ejector
- 6b secondary low pressure inlet port of the first controllable ejector
- 6c output port of the first controllable ejector
- 7 second controllable ejector
- 7a primary high pressure inlet port of the second controllable ejector
- 7b secondary low pressure inlet port of the second controllable ejector
- 7c outlet of the second controllable ejector
- 8 receiver
- 8a inlet of the receiver
- 8b gas outlet of the receiver
- 8c liquid outlet of the receiver
- 9 low temperature flowpath
- 10 refrigeration expansion device
- 10a inlet side of the refrigeration expansion device
- 10b outlet side of the refrigeration expansion device
- 12 refrigeration evaporator
- 12b outlet of the refrigeration evaporator
- 14 low temperature expansion device
- 16 low temperature evaporator
- 18 low temperature compressor unit
- 18a, 18b low temperature compressors
- 20

	service valve
21a, 21b, 21c	inlet side of the compressors
22a, 22b, 22c	outlet side of the compressors
26, 27	controllable valves at the secondary low pressure input ports
28	control unit
30	pressure and/or temperature sensor
31	high pressure input line
32	pressure and/or temperature sensor
33	low pressure input line
34	pressure and/or temperature sensor
35	ejector output line
36	receiver liquid outlet line
38	fan of the heat rejecting heat exchanger/gas cooler
40	receiver gas outlet line
100	motive nozzle
102	outer member
103	primary refrigerant flow
104	convergent section of the motive nozzle
106	throat section
108	divergent expansion section
110	outlet of the motive nozzle
112	secondary flow
114	

	convergent section of the mixer
116	
	throat or mixing section
118	
	diffuser
120	
	combined flow
130	
	needle valve
132	
	needle
134	
	actuator

REFERENCES CITED IN THE DESCRIPTION

Cited references

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P A T E N T K R A V

1. Fremgangsmåde til betjening af et ejektorkølekredsløb (1) med mindst to styrbare ejektorer (6, 7), der er forbundet parallelt og hver især omfatter en styrbar drivdyse (100), en primær højtryksindgangsport (6a, 7a), der danner en indløb til den styrbare drivdyse (100), en sekundær lavtryksindgangsport (6b, 7b) og en udgangsport (6c, 7c), hvor fremgangsmåden er **kendetegnet ved** at omfatte trinnene:

- a) betjening af en første ejektor (6) af de mindst to styrbare ejektorer (6, 7) ved at styre åbningsgraden af dens primære højtryksindgangsport (6a), indtil den maksimale effekt af den første ejektor (6) er nået eller de faktiske kølebehov er opfyldt,
- b) betjening af mindst én yderligere ejektor (7) af de mindst to styrbare ejektorer (6, 7) ved gradvist at åbne dens primære højtryksindgangsport (6a, 7a) for at øge kølekapaciteten af ejektorkølekredsløbet (1) i tilfælde af, at de faktiske kølebehov ikke opfyldes ved at betjene den første ejektor (6) alene.

2. Fremgangsmåde ifølge krav 1, hvor ejektorkølekredsløbet (1) yderligere omfatter:

- en varmeafvisende varmeveksler/gaskøler (4) med en indløbsside (4a) og en udløbsside (4b), idet udløbssiden (4b) af den varmeafvisende varmeveksler/gaskøler (4) er væskemæssigt forbundet til de primære højtryksindgangsporte (6a, 7a) i ejektorerne (6, 7),
- en modtager (8), der har et væskeudløb (8c), et gasudløb (8b) og et indløb (8a), der er væskemæssigt forbundet til udgangsportene (6c, 7c) i de styrbare ejektorer (6, 7),
- mindst én kompressor (2a, 2b, 2c) med en indløbsside (21a, 21b, 21c) og en udløbsside (22a, 22b, 22c), idet indløbssiden (21a, 21b, 21c) af den mindst ene kompressor (2a, 2b, 2c) er væskemæssigt forbundet til gasudløbet (8b) i modtageren (8), og udløbssiden (21a, 21b, 21c) af den mindst ene kompressor (2a, 2b, 2c) er væskemæssigt forbundet til indløbssiden (4a) af den varmeafvisende varmeveksler/gaskøler (4),
- mindst én køleekspansionsanordning (10) med en indløbsside (10a), der er væskemæssigt forbundet til væskeudløbet (8c) i modtageren (8), og en udløbsside (10b), og
- mindst én kølefordamper (12), der er væskemæssigt forbundet mellem udløbssiden (10b) af den mindst ene køleekspansionsanordning (10) og de sekundære lavtryksindgangsporte (6b, 7b) i de styrbare ejektorer (6, 7).

3. Fremgangsmåde ifølge krav 1 eller 2, hvor fremgangsmåden omfatter betjening af mindst én af de styrbare ejektorer (6, 7) med dens sekundære lavtryksindgangsport (6b, 7b) lukket.

4. Fremgangsmåde ifølge krav 3, der omfatter trinnet at åbne den sekundære lavtryksindgangsport (6b, 7b) i den mindst ene styrbare ejektor (6, 7), der er blevet betjent med dens sekundære lavtryksindgangsport (6b, 7b) lukket, hvor den sekundære lavtryksindgangsport (6b, 7b) især åbnes gradvist.

5. Fremgangsmåde ifølge et af de foregående krav, der omfatter trinnet at lukke den primære højtryksindgangsport (6a) og/eller den sekundære lavtryksindgangsport (6b) i den første ejektor (6).

6. Fremgangsmåde ifølge et af de foregående krav, der omfatter anvendelse af kuldioxid som kølemiddel.
7. Ejektorkølekredsløb (1), der er konfigureret til at cirkulere et kølemiddel, især kuldioxid, og omfatter:
- 5 mindst to styrbare ejektorer (6, 7), der er forbundet parallelt og henholdsvis omfatter en styrbar drivdyse (100), en primær højtryksindgangsport (6a, 7a), der danner en indløb til den styrbare drivdyse (100), en sekundær lavtryksindgangsport (6b, 7b) og en udgangsport (6c, 7c), og en styreenhed (28), kendetegnet ved, at styreenheden er konfigureret til at betjene ejektorkølekredsløbet (1) ved anvendelse af en fremgangsmåde, der omfatter trinnene:
- 10 a) betjening af en første ejektor (6) af de mindst to styrbare ejektorer (6, 7) ved at styre åbningsgraden af dens højtryksport (6a), indtil den maksimale effekt af den første ejektor (6) er nået eller de faktiske kølebehov er opfyldt,
- b) betjening af mindst én yderligere ejektor (7) af de mindst to styrbare ejektorer (6, 7) ved gradvist at åbne dens primære højtryksindgangsport (7a) for at øge kølekapaciteten af ejektorkølekredsløbet (1) i tilfælde af, at de faktiske kølebehov ikke opfyldes ved at betjene den første
- 15 ejektor (6) alene.
8. Ejektorkølekredsløb (1) ifølge krav 7, der yderligere omfatter:
- en varmeafvisende varmeveksler/gaskøler (4) med en indløbsside (4a) og en udløbsside (4b), idet udløbssiden (4b) af den varmeafvisende varmeveksler/gaskøler (4) er væskemæssigt forbundet til de primære højtryksindgangsporte (6a, 7a) i de styrbare ejektorer (6, 7),
- 20 en modtager (8), der har et væskeudløb (8c), et gasudløb (8b) og et indløb (8a), der er væskemæssigt forbundet til udgangsportene (6c, 7c) i de styrbare ejektorer (6, 7),
- mindst én kompressor (2a, 2b, 2c) med en indløbsside (21a, 21b, 21c) og en udløbsside (22a, 22b, 22c), idet indløbssiden (21a, 21b, 21c) af den mindst ene kompressor (2a, 2b, 2c) er væskemæssigt forbundet
- 25 til gasudløbet (8b) i modtageren (8), og udløbssiden (22a, 22b, 22c) af den mindst ene kompressor (2a, 2b, 2c) er væskemæssigt forbundet til indløbssiden (4a) af den varmeafvisende varmeveksler/gaskøler (4),
- mindst én køleekspansionsanordning (10) med en indløbsside (10a), der er væskemæssigt forbundet til væskeudløbet (8c) i modtageren (8), og udløbsside (10b), og
- 30 mindst én kølefordamper (12), der er væskemæssigt forbundet mellem udløbssiden (10b) af den mindst ene køleekspansionsanordning (10) og de sekundære lavtryksindgangsporte (6b, 7b) i de styrbare ejektorer (6, 7).
9. Ejektorkølekredsløb (1) ifølge krav 7 eller 8, hvor de styrbare ejektorer (6, 7) er tilvejebragt med samme
- 35 kapacitet.
10. Ejektorkølekredsløb (1) ifølge krav 7 eller 8, hvor de styrbare ejektorer (6, 7) er tilvejebragt med forskellige kapaciteter.

11. Ejektorkølekredsløb (1) ifølge et hvilket som helst af kravene 7 til 10, hvor en styrbar ventil (26, 27) er tilvejebragt opstrøms for den sekundære lavtryksindgangsport (6b, 7b) i mindst én/hver af de styrbare ejektorer (6, 7).

5 12. Ejektorkølekredsløb (1) ifølge et hvilket som helst af kravene 7 til 11, hvor mindst én sensor (30, 32, 34), der er konfigureret til at måle trykket og/eller temperaturen af kølemidlet, er tilvejebragt i mindst én af henholdsvis en højtryksindgangsledning (31), der er væskemæssigt forbundet til de primære højtryksindgangsporte (6a, 7a), en lavtryksindgangsledning (33), der er væskemæssigt forbundet til de sekundære lavtryksindgangsporte (6b, 7b), og en ejektorudgangsledning (35), der er væskemæssigt forbundet til udgangsportene (6c, 10 7c) i de styrbare ejektorer (6, 7).

13. Ejektorkølekredsløb (1) ifølge et hvilket som helst af kravene 7 til 12, hvor mindst en serviceventil (20) er tilvejebragt opstrøms for de styrbare ejektorers (6, 7) primære højtryksindgangsporte (6a, 7a).

15 14. Ejektorkølekredsløb (1) ifølge krav 13, der yderligere omfatter mindst et lavtemperaturkredsløb (9), der er forbundet mellem væskeudløbet (8c) i modtageren (8) og indløbssiden (21a, 21b, 21c) af den mindst én kompressor (2a, 2b, 2c) og i strømningsretningen af kølemidlet omfatter:

mindst én lavtemperaturrekspansionsanordning (14),
mindst én lavtemperaturfordamper (16), og
20 mindst én lavtemperaturkompressor (18a, 18b).

DRAWINGS

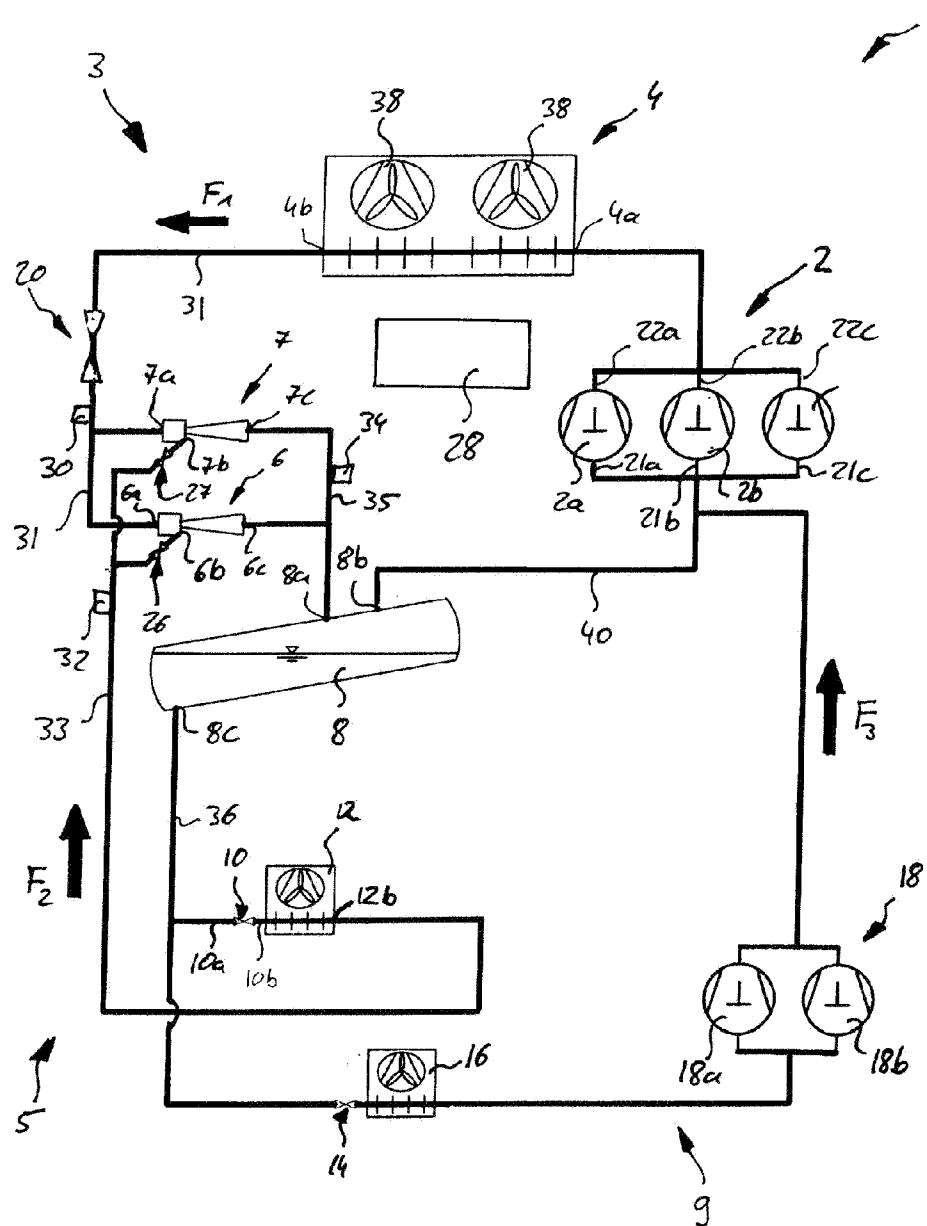


FIG.1

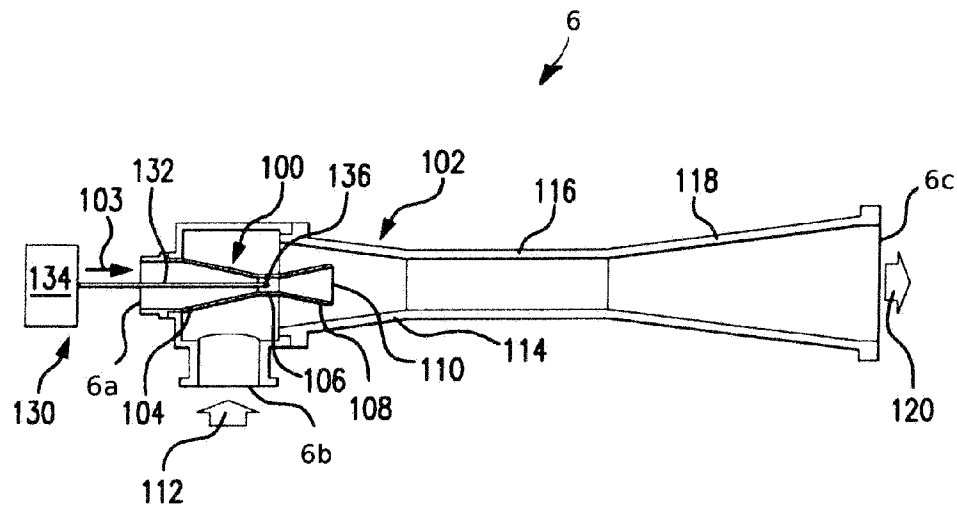


FIG. 2