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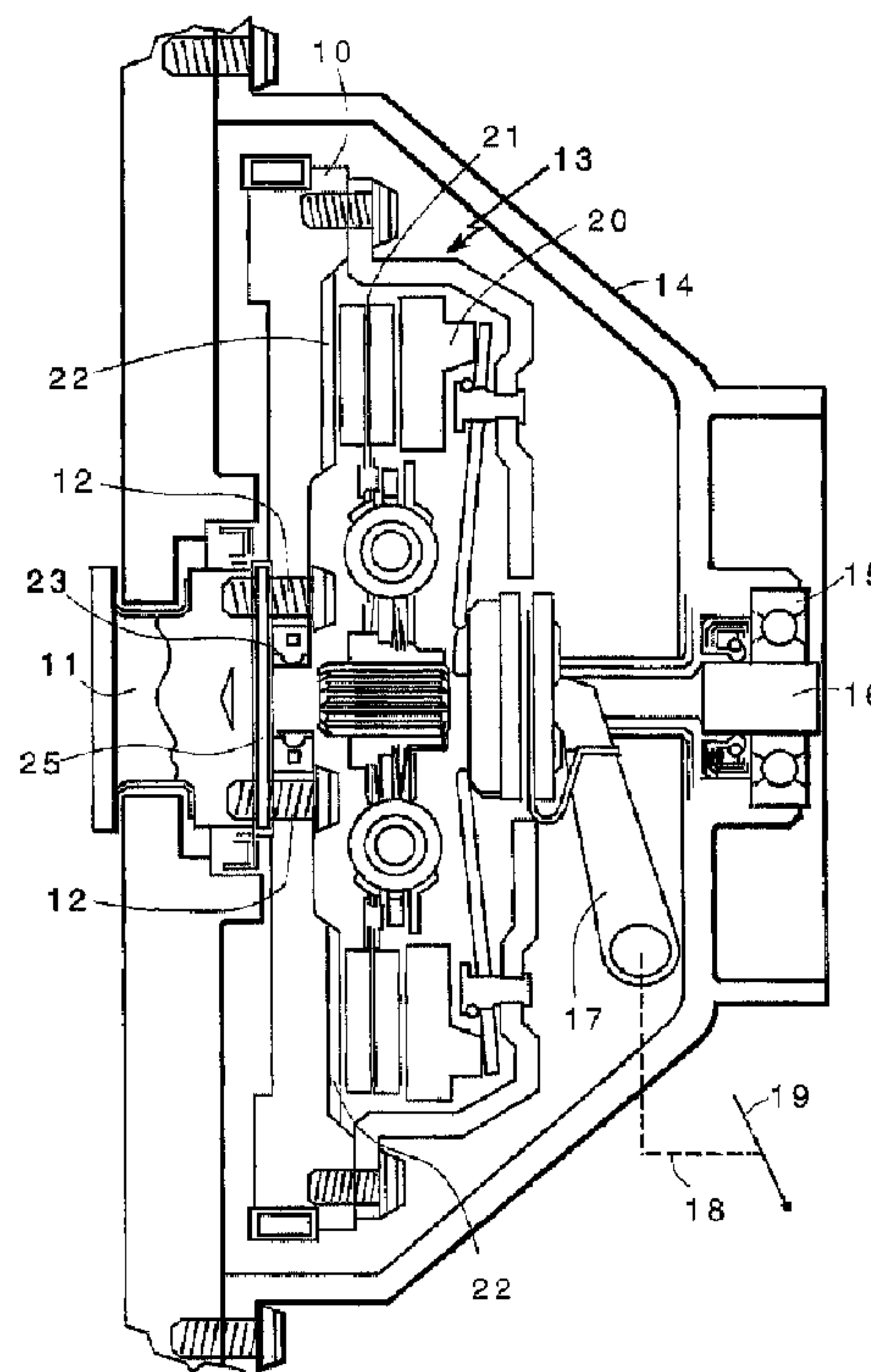
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(54) Title: SYSTEM FOR MAINTAINING CLUTCH PEDAL HEIGHT AFTER CLUTCH RESURFACING



(57) Abrégé/Abstract:

Standard internal combustion engine transmission system having an engine crankshaft, clutch, flywheel connecting the engine to a drive gear box, a coupling arbor hub having a planar face and a complementary planar face on the flywheel, and a plurality of bolts securing the flywheel to the arbor hub with the planar faces abutting each other. The flywheel has a clutch surface which can be resurfaced by machining to extend the life of said flywheel. The invention reestablishes the substantially the same position of the clutch surface after machining as when the clutch surface was new, It includes one or more planar annulus plates having a pair of parallel surfaces spaced apart by the thickness T of the planar annulus plate member, the thickness T being substantially the amount of metal removed during the machining of said surface, said annulus plate member having a plurality of bolt holes, one for each of the bolts to pass therethrough, respectively.



SYSTEM FOR MAINTAINING CLUTCH PEDAL
HEIGHT AFTER CLUTCH RESURFACING

ABSTRACT

Standard internal combustion engine transmission system having an engine crankshaft, clutch, flywheel connecting the engine to a drive gear box, a coupling arbor hub having a planar face and a complementary planar face on the flywheel, and a plurality of bolts securing the flywheel to the arbor hub with the planar faces abutting each other. The flywheel has a clutch surface which can be resurfaced by machining to extend the life of said flywheel. The invention reestablishes the substantially the same position of the clutch surface after machining as when the clutch surface was new, It includes one or more planar annulus plates having a pair of parallel surfaces spaced apart by the thickness T of the planar annulus plate member, the thickness T being substantially the amount of metal removed during the machining of said surface, said annulus plate member having a plurality of bolt holes, one for each of the bolts to pass therethrough, respectively.

SYSTEM FOR MAINTAINING CLUTCH PEDAL HEIGHT AFTER CLUTCH RESURFACING

Field and Background of the Invention

The present invention is in the field of flywheel resurfacing and relates to the problem
5 of maintaining the clutch surface position after resurfacing and has as its object the avoidance
of costly replacement of the flywheel when too much metal is removed from the flywheel or
after multiple resurfacing operations on the same flywheel.

Clutch flywheels are frequently resurfaced to remove surface glazing, roughness and
other detriments to normal transmission operation. If too much metal is removed the clutch
10 release fork geometry can be changed to significantly interfere with normal operation and
affect the clutch pedal position resulting in customer dissatisfaction. The solution in the past
has been to replace the flywheel at a cost of several hundred dollars. Other potential spacing
problems solved by the invention include excessive metal removed during pump rebuilding,
excessive variation in rebuilt torque converter height and main thrust bearing wear.

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Brief Description of the Invention

According to the present invention, there is provided an annular plate for use in an
internal combustion engine transmission system having an engine crankshaft, clutch, clutch
operating pedal and clutch release fork coupled to said clutch, flywheel connecting the engine
20 to a drive gear box, a coupling arbor having a planar face, a complementary planar face on
the flywheel, and a plurality of bolts securing the flywheel to the arbor with the planar faces

abutting each other. Within the concept of the invention the flywheel has a metal clutch surface which has been resurfaced by machining to remove a layer of metal means to re-establishing substantially the same position of the clutch surface after machining as when the clutch surface was new so that the operation of the clutch operating pedal is not affected by the machining. The planar annular plate of the invention is formed for location between the planar face and the complementary face, the planar annular plate having a crankshaft arbor aperture, a pair of parallel surfaces spaced apart by the thickness (T) of the planar annular plate member. The thickness (T) is substantially the amount of metal removed during the machining of said surface, the planar annular plate being adaptable to a plurality of different crankshaft arbors.

More specifically the planar annular plate has a crankshaft arbor aperture and a plurality of breakaway tabs positioned in the crankshaft arbor aperture for adapting the planar annular plate to different crankshaft arbor diameters.

According to one feature of the invention, there are provided a plurality of the annular plates in a range of thickness, and at least a pair of the annular plates are used to substantially provide the thickness (T).

An embodiment of the invention resides in the concept of utilizing a shim means for use in an internal combustion engine transmission system having an engine crankshaft, clutch, a metal flywheel connecting the engine to a drive gear box, a coupling arbor having a planar face and a complementary planar face on the flywheel, and a plurality of holes securing the flywheel to the arbor with the planar faces abutting each other, the flywheel having a metal clutch surface which has been resurfaced by machining to remove metal from the clutch

surface and extend the life of the flywheel. The shim means is provided for establishing substantially the same position of the clutch surface after machining as when the clutch surface was new, and the shim means includes a planar annular plate having a pair of parallel surfaces spaced apart by the thickness (T) of the planar annular plate member. The thickness
5 (T) being substantially the amount of metal removed during the machining of the surface.

The annular plate member has a plurality of bolt holes, one for each of the bolts to pass therethrough, respectively, with one of the pair of parallel surfaces being adapted to abut the planar face of the arbor and the other of the parallel surfaces being adapted to abut the complementary planar face of the flywheel. In the present invention, the annular plate has an
10 inner aperture with a plurality of breakaway tabs on the inner aperture which are adapted to be broken away to accommodate larger diameter crankshaft hubs.

Thus, it may be seen that instead of replacing the flywheel, the present invention provides a planar shim member which in its preferred embodiment is adaptable to a plurality of different standard transmissions. The plurality of break-
15 off centering tabs may be formed integral with the shim body and the plurality of bolt holes are formed to accommodate various engine arbor/flywheel bolt positions.

In this way, the invention solves the clutch release problem on standard transmissions, eliminates potential master cylinder problems, re-establishes proper clutch release fork geometry, avoids the cost of replacing the flywheel and re-establishes the original surface
20 position of the O.E.M. flywheel so that the clutch pedal returns to the original position.

Description of the Drawings

The above and other objects, advantages and features of the invention will become more apparent when considered with the following specification in conjunction with the accompanying drawings wherein:

5 Figure 1 is a sectional view of a standard transmission assembly incorporating the invention,

Figure 2 is a broad illustration of one embodiment of an planar annular plate of the invention,

10 Figure 3 is an illustration of an embodiment of the invention utilizing a pair of break-off tabs,

Figure 4 is an illustration of yet another embodiment of the invention showing another form of break-off tabs and a modified form of bolt holes, and

Figure 5 is an illustration of a further embodiment of the invention utilizing two different series of break-off tabs.

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Detailed Description of the Invention

In an internal combustion driven vehicle having a standard transmission, power from the engine is coupled to the wheels by a friction clutch mechanism incorporating a flywheel having a clutch surface which is frictionally engaged by a clutch plate drive to engagement
20 with the clutch surface by a clutch pedal via a linkage and a clutch release fork or yoke.

When the clutch requires service, frequently this involves resurfacing the clutch surface and after multiple resurfacing, the geometry is such that the spacing between the flywheel surface

and the clutch plate is changed sufficiently to affect the performance and position and extent of pedal movement required to engage the clutch. This lead to numerous customer complaints about the quality of service when the service may have been exemplary. The present invention solves this problem in a expeditious and cost saving manner.

5 Referring now to Figure 1, a standard flywheel 10 is secured to the engine (not shown) by an engine crank shaft arbor/hub 11 by bolts 12. The clutch assembly 13 is protected by bell housing 14 which carries bearings 15 for clutch shaft 16 which, in turn, is coupled to a transmission mechanism (not shown). The clutch assembly 13 is slidable along clutch shaft 16 by clutch release fork 17 which is coupled by a linkage 18 to clutch pedal 19.

10 Clutch plate 20 carries friction members 21 which engage clutch surface 22 on flywheel 10.

As described earlier, when the clutch requires service, in addition to replacing the friction members 21, the clutch surface 22 frequently has to be machined to restore it and after multiple resurfacing, the spacing between the clutch surface 22 and the rest position of the friction members on the clutch plate is such that the clutch pedal must be depressed to a

15 great extent before there is engagement and in the past this has been solved at great expense and dissatisfaction to the customer by replacing the entire flywheel.

The present invention solves this problem by inserting a planar annular plate or shim 25, as shown in Figure 2, between the engine arbor 11 and flywheel 10, the thickness of shim 25 is a function of the member of resurfacing but is preferably provided in sets of stock sizes

20 such as about .025", .040", .089", etc. Combinations of shim 25 sizes can be used. For example a .025" thick shim can be used with a .040" shim to provide in effect a .065" shim.

Since the bell housing and flywheel have been removed for servicing, the installation

of shim 25 does not require more work or installation time in servicing the clutch and results in far fewer customer complaints.

In order that the shim, which is formed by one or more of the planar annular plate members 25, be adaptable to different diameters of engine arbor-clutch shaft sizes, the preferred embodiment of the planar plate member includes a plurality of break-off tabs 30, as shown in Figure 3. Each break-off tab 30 is formed in an inner aperture 24 defined within the inner circular edge S1. Each tab 30 as shown in Figure 3, is joined to a main body portion of the planar annular plate member 25' by breakaway sections which may include a score line or a weakened portion 32. It will be noted that each tab has an arcuate outer surface S2 to accommodate the clutch shaft bearings 23 (Fig. 1). Moreover the breakaway portions 32 are recessed below the circular edges S1 so that after the breaking away of tabs 30, the remaining parts of break away sections 32 are at or below circular edge S1 of the inner opening or aperture 24. In the embodiments of Figures 3 and 4, two different diameters are accommodated, i.e., either by the surface S2 of the tab or the circular edge S1 of the aperture 24. In Figure 4, the break-off tabs consist of three separate tabs, 30', 30', 30' defining inner surfaces S2', S2', S2' of smaller diameter than the inner periphery S1' of the opening. In the embodiment of Figure 5 there are two sets of break-off tabs which enables three diameters of shaft to be accommodated. That is, the outer arcuate surfaces S2 and S3 of tabs 38 and 39 are on different radii. Instead of being spaced circumferentially the breakaway tabs can be aligned. Bolt holes BH and a KEY notch (Figure 2) are spaced to accommodate the bolts 12. In Figure 4 the bolt holes BH'' are elongated and extend into the inner annular space. In the embodiment of Figure 5 bolt holes BHK are shaped differently than holes BH'''.

While the invention has been described and illustrated in respect to its preferred embodiments, it will be appreciated that various other embodiments and adaptations of the invention will occur to those skilled in the art.

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CLAIMS

1. In an internal combustion engine transmission system having an engine crankshaft, clutch, a metal flywheel connecting the engine to a drive gear box, coupling arbor having a planar face and a complementary planar face on said flywheel, and a plurality of bolts securing said flywheel to said arbor with said planar faces abutting each other, said flywheel having a metal clutch surface which has been resurfaced by machining to remove metal from said clutch surface of said flywheel, shim means for establishing substantially the same position of said clutch surface after machining as when the clutch surface was new, said shim means comprising a planar annular plate member having a pair of parallel surfaces spaced apart by the thickness (T) of the planar annular plate member, said thickness (T) being substantially the amount of metal removed during the machining of said surface, said annular plate member having a plurality of bolt holes, one of each of said bolts to pass therethrough, respectively, one of said pair of parallel surfaces being adapted to abut said planar face on said arbor and the other of said parallel surfaces being adapted to abut said complementary planar face on said flywheel, the improvement wherein said annular plate member has an inner aperture, a plurality of breakaway tabs in said inner aperture which are adapted to be broken away to accommodate larger diameter crankshaft hubs.

2. In an internal combustion engine transmission system having an engine crankshaft, clutch, clutch operating pedal and clutch release fork coupled to said clutch, flywheel connecting the engine to a drive gear box, a coupling arbor having a planar face, a complementary planar face on said flywheel, and a plurality of bolts securing said flywheel to said arbor with said planar faces abutting each other, said flywheel having a metal clutch surface which has been resurfaced by machining to remove a layer of metal means to reestablishing substantially the same position of said clutch surface after machining as when the clutch surface was new so that the operation of said clutch operating pedal is not affected by said machining, comprising a planar annular plate

between said planar face and said complementary face, said planar annular plate having a crankshaft arbor aperture, a pair of parallel surfaces spaced apart by the thickness (T) of said planar annular plate member, said thickness (T) being substantially the amount of metal removed during the machining of said surface, said planar annular plate being adaptable to a plurality of different crankshaft arbors.

3. The engine transmission system defined in claim 2 wherein said planar annular plate has a crankshaft arbor aperture and a plurality of breakaway tabs positioned in said crankshaft arbor aperture for adapting said planar annular plate to different crankshaft arbor diameters.

4. The engine transmission system defined in claim 2 wherein there are provided a plurality of said annular plates in a range of thickness, and at least a pair of said annular plates are used to substantially provide said thickness (T).

5. The transmission system defined in claim 3 wherein there are least two sets of said breakaway tabs, each breakaway tab of a set having the same diameter with each set having a different diameter.

6. In an internal combustion engine transmission system having an engine crankshaft, a metal flywheel connecting the engine to a drive gear box, a coupling arbor having a planar face and a complementary planar face on said flywheel, a plurality of bolts securing said flywheel to said arbor with said planar faces abutting each other, said flywheel having a transmission surface which has been resurfaced by machining to remove a layer of metal and extend the life of said flywheel, the improvement for reestablishing substantially the same position of said transmission surface in said transmission system after machining as when the transmission surface was new, comprising a planar annular plate

having a pair of parallel surfaces spaced apart by the thickness (T) of said planar annular plate member, said thickness (T) being substantially the amount of metal removed during the machining of said transmission surface, said annular plate member being adapted to have one of said pair of parallel surfaces abut having a plurality of bolt holes, one for each of said bolts to pass therethrough, respectively, the improvement comprising a plurality of spaced arcuate breakaway tabs positioned in said central aperture for adapting said shim means to different diameter arbors.

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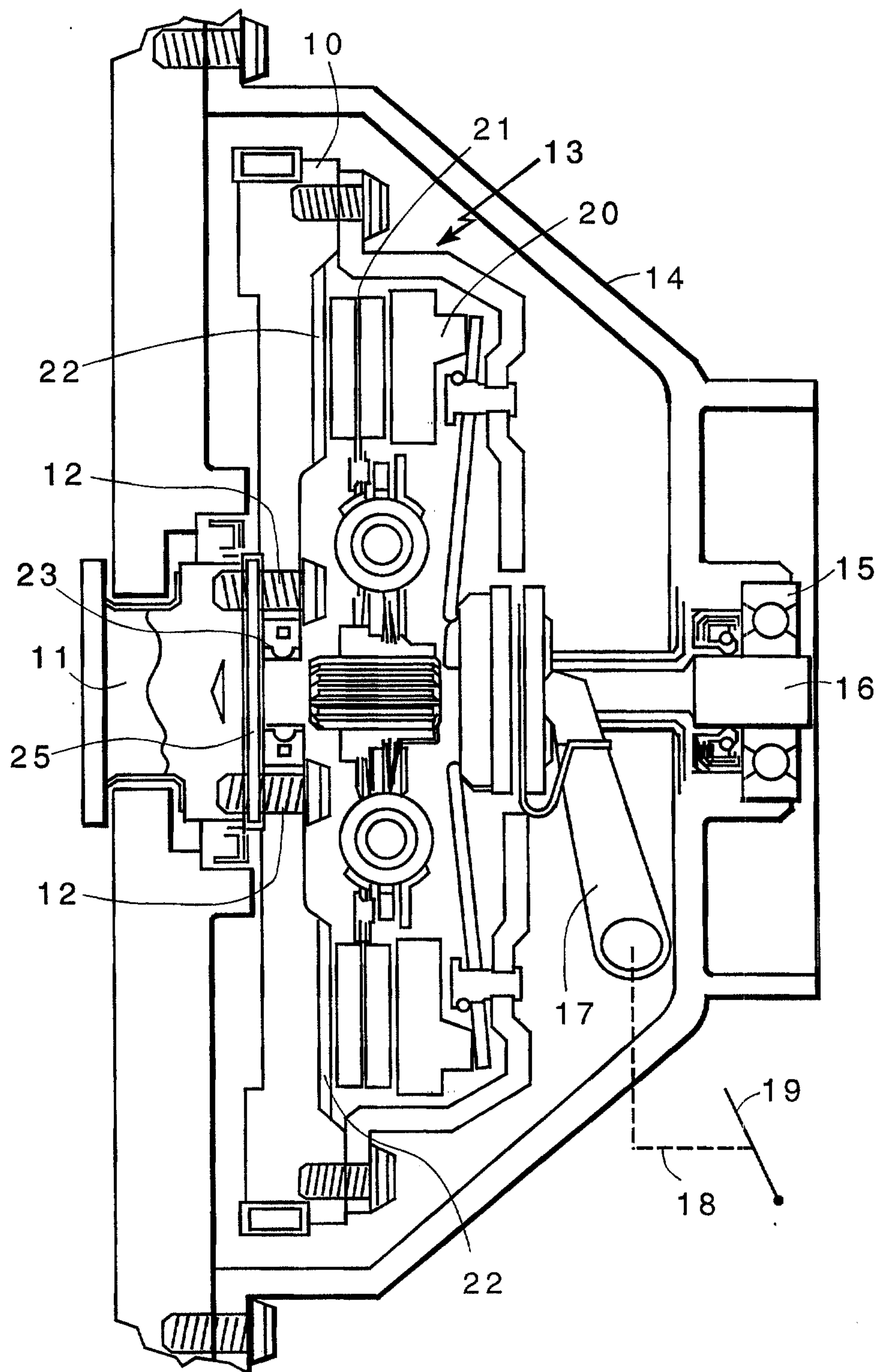


FIGURE 1

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FIGURE 2

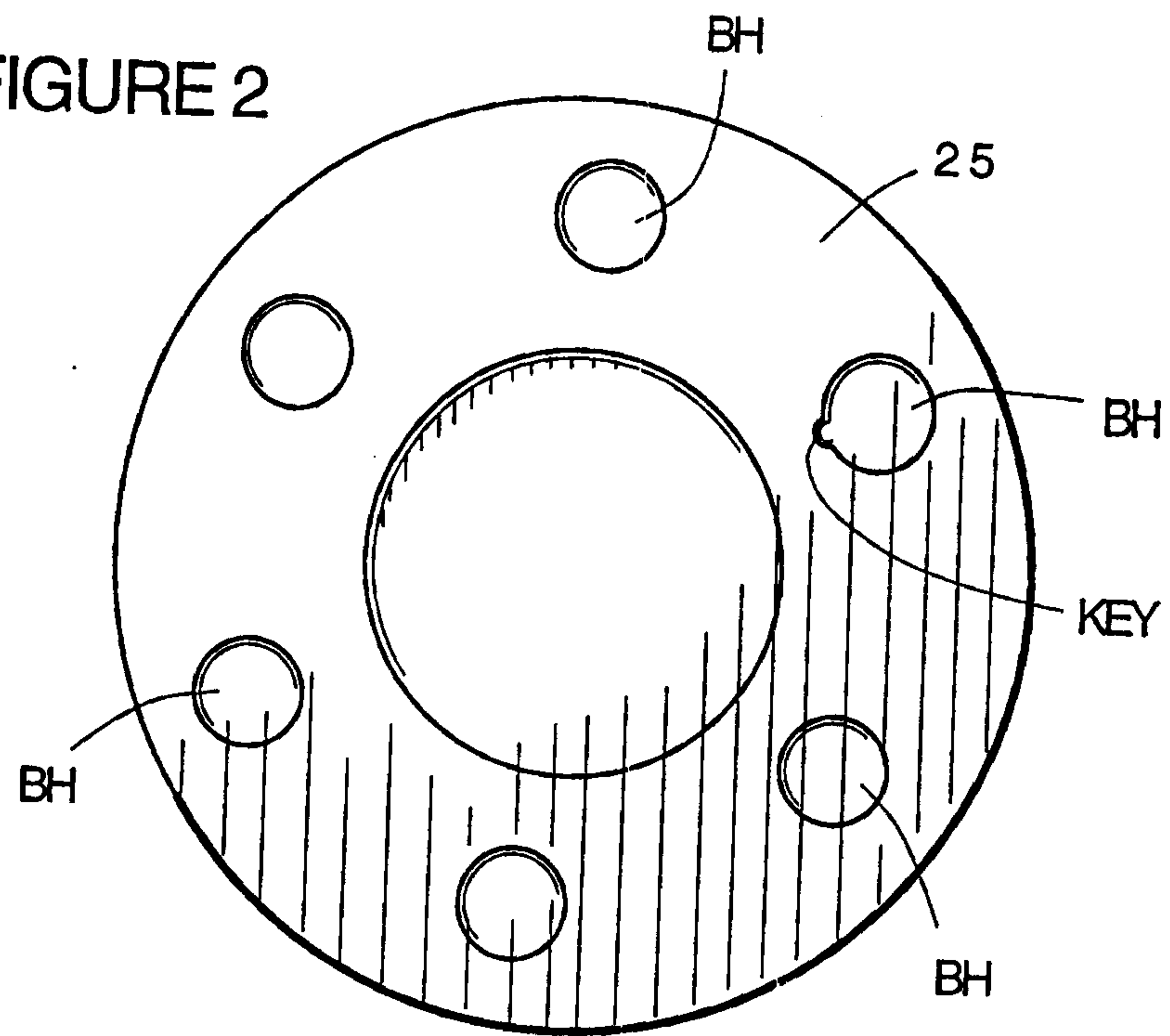
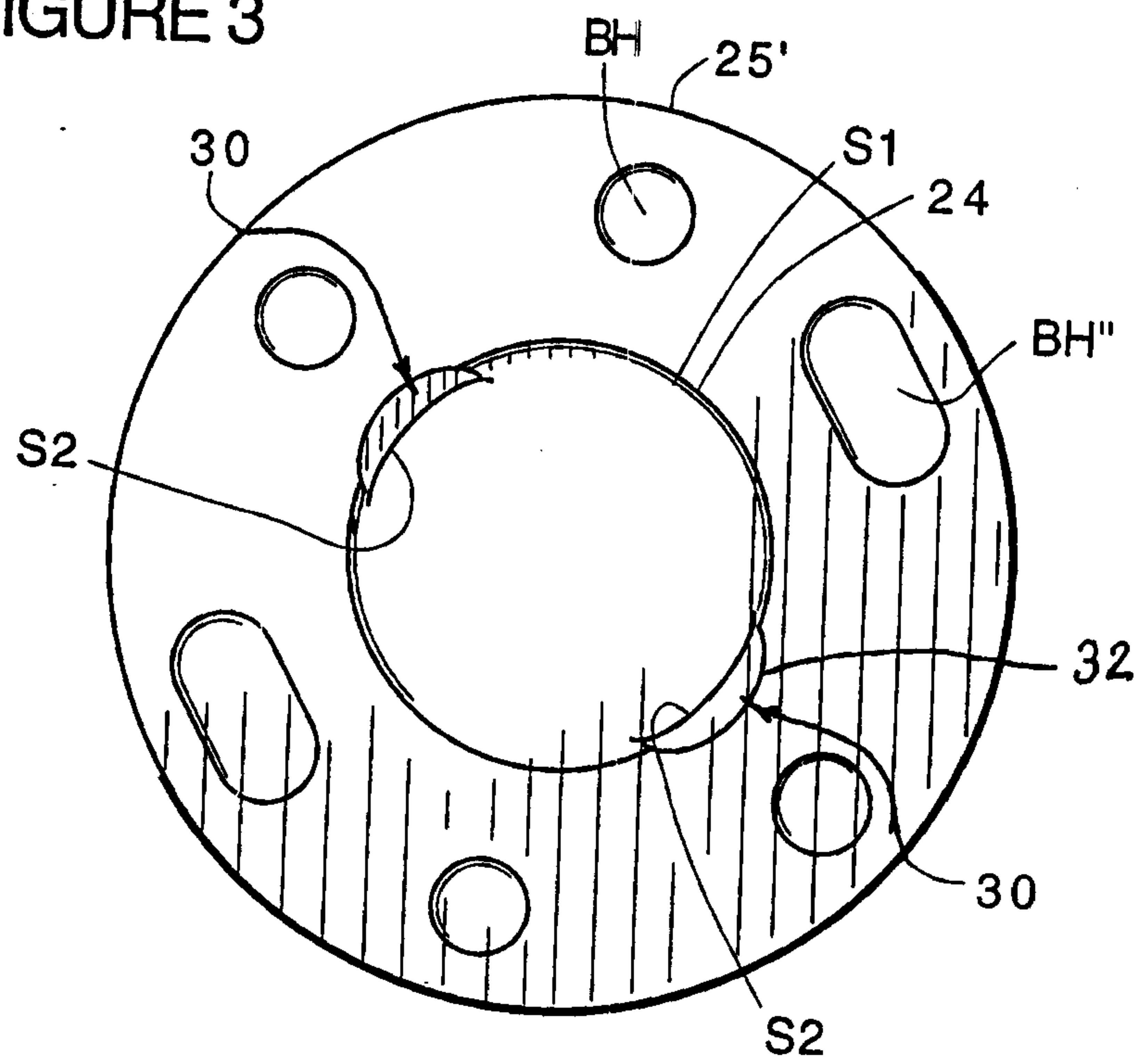


FIGURE 3



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FIGURE 4

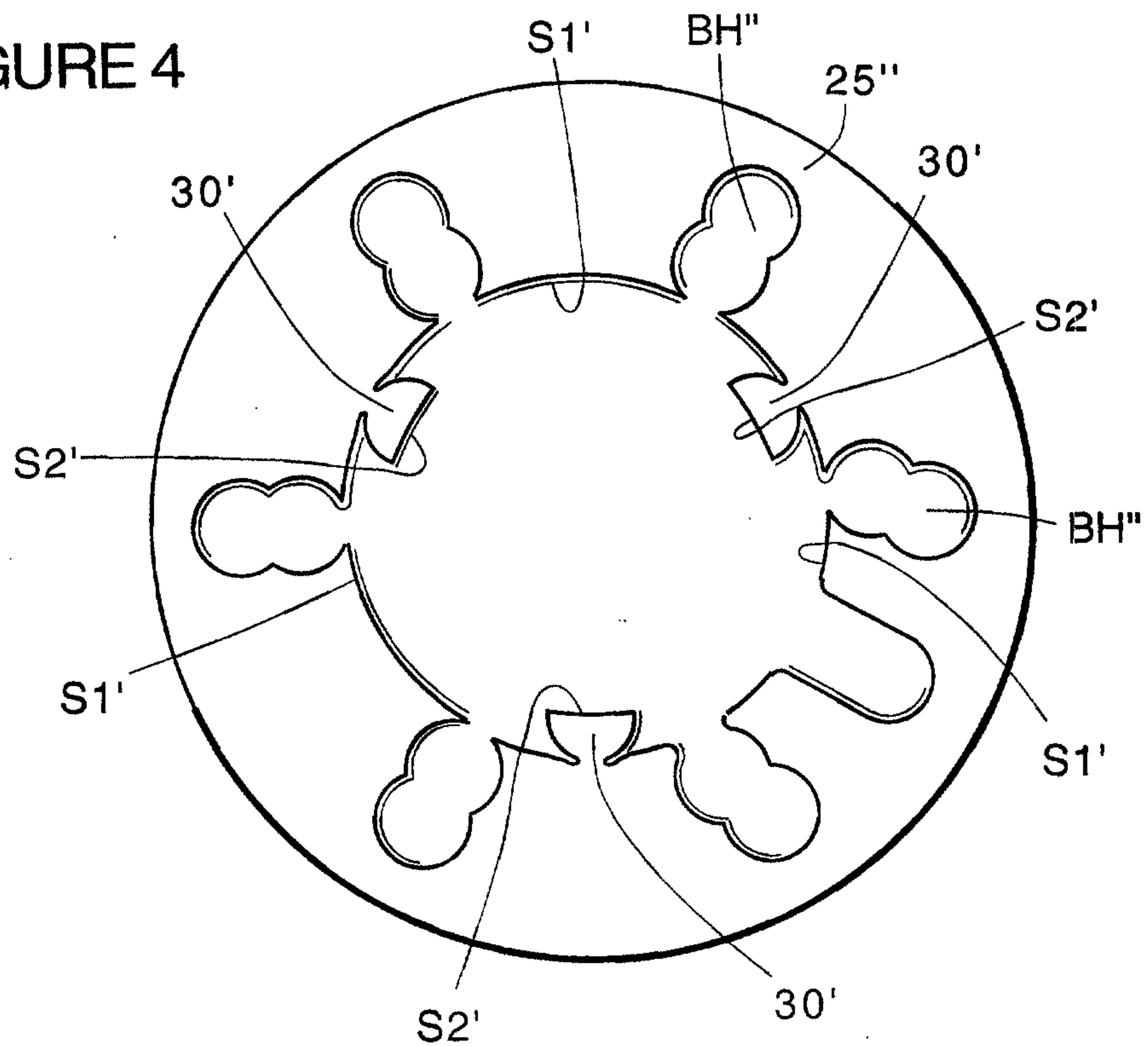


FIGURE 5

