



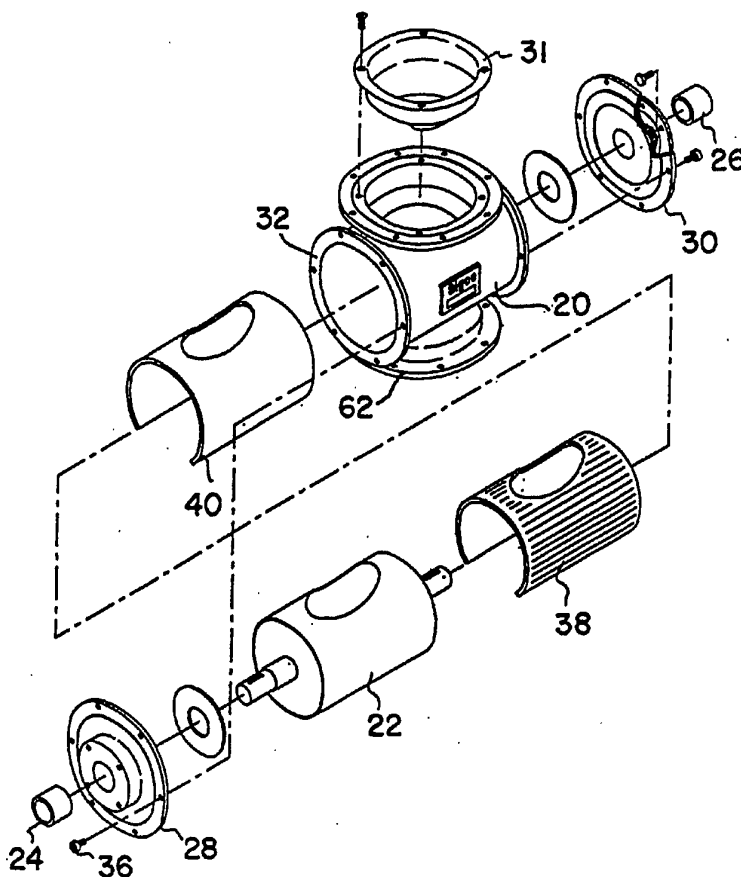
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(54) Title: ROTARY AIRLOCK VALVE USING A SINGLE SEAT

(57) Abstract

A vaneless rotor airlock valve using a single seat. A hollow rotor (22) is supported by two bearings (24, 26) to rotate within a first bore of the valve housing (20), which includes intersecting transverse bores. The rotor (22) includes an opening which extends a selected distance of less than 90 degrees in a plane perpendicular to the axis. The ends of the second bore terminate at the input port and the output port of the valve. Sealing contact is maintained between the outer surface of the rotor and a non-metallic seat member (38) to maintain pressure isolation between the input port and the output port as the rotor (22) is rotated to alternately position the rotor (22) in a fill and discharge position. All portions of the interior of the valve are free draining under the force of gravity.



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Rotary Airlock Valve Using a Single Seat

Description of Prior Art

Vaneless Rotor Airlock Valves are well known in the prior art. For example Vaneless Rotor Airlock Valves are commercially available from Sigco Valve Company, Charlotte, North Carolina. 5 These valves function for their intended purpose. However, these valves typically include entrapment zones between the exterior of the rotor and the interior of the housing, in which portions of the material being transferred through the valve may accumulate. (An entrapment zone is defined as a space in which 10 material being transferred through the valve may become entrapped such that it is difficult to remove without disassembling the valve.) Material accumulating in these zones cannot be easily removed. This feature is undesirable in selected applications, such as some food handling applications. Furthermore, these 15 prior art valves utilized separate seats on the input and the output ports. This feature added significantly to the cost of these valves.

Description of the Invention

The preferred embodiment of the invention comprises a vaneless 20 rotor airlock valve without entrapment zones between the interior of the housing and the exterior of the rotor. A housing including intersecting transverse bores supports a rotor, which includes a cavity, in the first bore and concentric thereto. The ends of the second bore defining the input port and the output 25 port of the valve. Pressure sealing is provided between the input port and the output port by a sealing structure (including a seat and a compression seal) positioned between the outer surface of the rotor and the interior surface of the first bore. The preferred embodiment of the sealing structure includes a 30 seat, formed of a flexible material in the shape of a cylinder with a gap parallel to its axis.

In addition to eliminating the entrapment zones associated with prior art Vaneless Rotor Airlock Valves, the valve that is the subject matter of this invention utilizes one seat (described above) for both the input port and the output port. This feature
5 significantly reduces the manufacturing cost and improves the performance by eliminating the entrapment zones associated with prior art rotary airlock valves.

The invention may include a material control adapter for controlling the flow of material into the rotor of a vaneless
10 rotor air lock valve which includes a bottom end adapted to mate with the rotor of said vaneless rotor air lock valve, an upper end having a surface adapted to mate with the input flange of said vaneless rotor air lock valve, and a side portion extending
15 between said bottom end and said upper end to form a passageway through which material being transferred through said vaneless rotor airlock valve flows. The dimensions of said bottom end are selected to provide a selected distance between the outer surface of said bottom end and the edge of the entry way in the rotor seal of said vaneless rotor airlock valve. Specifically the
20 invention is referred to as an adapter in that it can be used with all know versions of vaneless rotor air lock valves.

Description of the Drawings

Figure 1 is an exploded view of the preferred embodiment of the vaneless rotor airlock valve, comprising the preferred embodiment
25 of the invention.

Figure 2 is a top view of the seat used by the preferred embodiment of the invention.

Figure 3 is a cross section view of the seat along section line A'-A' of Figure 2.

30 Figure 4 is a cross section view of the seat along section line B'-B' of Figure 2.

Figure 5 is a cross section view of the preferred embodiment of the invention in a plane perpendicular to the axis of rotation of the rotor and coincident with the axis of the input port and

the output port.

Figure 6 is an end view of the preferred embodiment of the invention with the end plate removed.

Figure 7 is a cross section view of one embodiment of the rotor.

5 Figure 8 is a cross section view of a second embodiment of the rotor.

Figure 9 is an exploded view of a second embodiment of the invention.

10 Figures 10, 11, and 12, are respectively section and end views of a second seat.

Figure 13 is a plan view of a second embodiment of the compression seal.

Figure 14 is a cross section of the second embodiment of the invention with the rotor in the fill position.

15 Figure 15 is a cross section view of a third embodiment of the the top, cross invention in a plane transverse to the axis of the rotor.

Figure 16 is a cross section view of the third embodiment of the invention in the plane parallel to the axis of the rotor.

20 Figure 17 is a top view of the third embodiment of the invention.

Figure 18 is the bottom view of the third embodiment of the invention.

Figure 19 is a cross section view of a first embodiment of the rotor used in the third embodiment of the invention.

25 Figure 20 is a cross section view of a second embodiment of the rotor used in the third embodiment of the invention.

Figures 21, 22, and 23, are respectively top, cross section and end views of the seat used in the third embodiment of the invention.

30 Figure 24 is a plan view of the compression seal used in the third embodiment of the invention.

Figure 25 is a drawing in cross section illustrating a vaneless rotor air lock valve utilizing a material control adapter.

Description of the Invention

35 The preferred embodiment of the invention is illustrated in

exploded view in Figure 1. A housing 20 provides means for mounting all components of the vaneless rotor airlock valve in the proper relationship to each other.

Specifically, the rotor 22 is supported in a first bore, of a
5 housing 20 having transverse bores, by first and second bearings, 24 and 26, permitting the rotor 22 to be rotated about its axis by drive means, not shown.

Means for supporting the bearings, 24 and 26, is provided by two
10 end plates, 28 and 30. Identical flanges, typically illustrated at reference number 32, mate with the end plates, 28 and 30. The end plates, 28 and 30, are secured to the housing 20 with fasteners, typically illustrated at reference number 36.

A seat 38 is mounted around the rotor 22 such that the inner
15 surface of the seat 38 is in a sealing relationship to the outer surface of the rotor 22. External pressure is applied to the outer surface of the seat 38 by a resilient compression seal 40, typically made of silicone foam. During normal use the ends of the seat 38 are held in a spaced apart relationship to form a gap, as illustrated in Figure 1.

20 The preferred embodiment of the seat 38 is respectively illustrated in top, cross section along its axis, and in cross section parallel to its axis, in Figures 2, 3 and 4. As illustrated in Figure 4, the seat 38 is semicircular in cross section and extends around the rotor 22 a radial distance in the
25 range of 270 degrees. That is to say, the gap is in the range of ninety degrees wide.

During normal operation, material being transferred through the valve is discharged through the gap between the ends of the seat
30 38. An elliptical opening 42 having a radial width of 90 degrees is equally positioned between the ends of the seat 38. Serrations are spaced along the exterior surface of the seat 38.

In the preferred embodiment, the seat is made of a plastic material, such as UHMW (ultra-high molecular weight) polyethylene. The various features of the seat 38 and those features common to an alternate embodiment of the seat are subsequently discussed in more detail.

Operation of the vaneless rotor airlock valve can best be understood with reference to Figure 5. In this drawing the assembled valve is illustrated in cross section view. From this drawing it can be seen that the seat 38 and the compression seal 40 are held in a fixed radial relationship with the other parts of the vaneless rotor airlock valve by a seat retainer 50 that extends downward into the opening in the seat 38 and the compression seal 40. Means holding the seat retainer 50 in a fixed relationship to the housing 20 of the valve is provided by suitable fastening means, such as screws 52.

Material to be transferred enters the vaneless rotor airlock valve through the input port which terminates in the input flange 60 and is collected in the cavity of the rotor 22. Drive means rotates the rotor 22 such that the opening in the rotor 22 is downward. In this position the material collected in the cavity of the rotor 22 is discharged by gravity through the output port terminating in output flange 62.

For a better understanding of the features of the preferred embodiment, the preferred embodiment is illustrated in end view with the end plate removed in Figure 6. Alternate embodiments of the rotor are also illustrated in Figures 7 and 8.

A first alternate embodiment of the invention is illustrated in Figure 9. In order to emphasize the similarity to this embodiment with the first and the preferred embodiment illustrated in Figure 1, the same reference numbers have been used to identify parts common to both embodiments.

The second embodiment of the invention illustrated in Figure 9,

differs from the embodiment illustrated in Figure 1, in that this embodiment utilizes a different seat 70 and a different compression seal 72. Additionally, a second seat retainer 35 extends upward into a second opening in the seat 70 on the
5 discharge side of the valve.

This embodiment has the characteristic that it removes ledges between the rotor 22 and the housing 20 at the output port of the valve. These ledges result from the longitudinal gap in the seat which extends outward beyond the inner surface of the housing.
10 While these ledges may not form entrapment zones, as the term was previously defined, they do provide spaces in which material may accumulate. Additional cleaning techniques may be required to remove such accumulation of the material being transferred.

Figures 10, 11, and 12, respectively illustrate the seat 70 in
15 top, cross section and end view. Identical top and bottom openings are provided for mating with the input and output seat retainers, 31 and 35. As with the first embodiment of the invention previously discussed, the compression seal 76 applies a force to the outer surface of the seat 70 to maintain the seat
20 70 in sealing contact with the outer surface of the rotor 22.

In operation, it is contemplated the rotor 22 will be driven in a forward-reverse cycle to sequentially position the rotor 22 in the fill and the discharge positions. When so driven, the gap 74 will never be exposed to the interior of the rotor 22,
25 preventing portions of the material being transferred from accumulating in the gap. Additionally, the seat 70 always provides a pressure seal isolating all interior portions of the valve from the cavity in the rotor 22. In this manner the shelves previously discussed with reference to Figure 1 are
30 eliminated in that the longitudinal gap 74 in the seat 70 is not normally exposed to the cavity of the rotor 22.

The compression seal 76 may be made of silicone sponge. This compression seal is illustrated in plan view in Figure 13. The

second embodiment of the invention is further illustrated in Figure 14. In this Figure the rotor 22 is illustrated in the fill position. In the fill position material to be transferred flows into the cavity of the rotor 22. Drive means, not
5 illustrated, drives the rotor to the discharge position with the material flowing downward under the force of gravity through the discharge port. Rotation of the rotor is reversed to return the rotor 22 to the fill position without exposing the gap 74 to the interior of the rotor 22, as previously discussed.

10 The seats, 38 and 70, contain common features which will now be collectively described. Serrations in the outer surface of the seat prevent the force applied to the outer surface of the seat by the compression seal from significantly deflecting portions of the seat into the opening in the rotor. Additionally, the
15 preferred material for the seat is a plastic. Such materials have a high coefficient of thermal expansion. Provisions are provided by the longitudinal gap in the seat to prevent thermal expansion from deforming the seat, such that the seat lifts from the surface of the rotor. This gap in the seat also permits the
20 seat to flex to compensate for rotor wear, seat wear or a combination thereof. The flutes in the outer surface of the seat permits the flexibility of the seat to be greater in a radial direction than it is in a longitudinal direction. Thus, the seat can be held in a sealing relationship with the rotor by the
25 compression seal without significant inward deformation over the opening in the rotor.

Alternate embodiments of the rotor have also been shown. These embodiments differ in that the trunnions have been reversed. Reversing the trunnions eliminates ledges interior to the rotor
30 that may undesirably accumulate material, in some applications.

Specifically, in Figure 7, the interior cavity 51 of the trunnion 61 faces outward to provide an end wall adjacent the opening in the rotor. This improves drainage from the rotor. By contrast, the rotor illustrated in Figure 8, the cavity of the trunnion 61

faces inward. This increases the capacity of the rotor with a slight decrease in drainage efficiency.

A third embodiment of the invention that can be easily fabricated from sheet materials is illustrated in cross section in Figures 5 15 and 16. Functionally, this embodiment contains all of the basic components discussed with respect to at least one of the prior embodiments. However, many of the components have been modified permitting them to be easily fabricated from sheet materials. These modifications also remove all of the ledges 10 which could be present between portions of the rotor and the housing in some of the previous embodiments.

As in the previous embodiments, a rotor 80 having an opening in its outer surface is supported in a housing 82 by end plates, 84 and 86, and bearings, 88 and 90, to rotate about its axis. A 15 seat 92 is positioned in sealing contact with the outer surface of the rotor 80. An inward directed pressure is applied to the outer surface of the seat 92 to maintain the seat 92 in sealing contact with the rotor 80. A seal retainer 93 that mates with the input flange 95, is secured to the housing 82 by screws, 20 extends through the compression seal 94 and a predetermined distance into the seat 92 such that a small clearance is maintained between the end of the seat retainer 93 and the surface of the rotor 80. The salient feature of third embodiment that permits the valve to be "free draining" is the use of 25 rectangular output port. These features are discussed in more detail below.

In operation, the material to be transferred through the valve enters the input port which terminates in the input flange 95, the rotor 80 is rotated through ninety degrees and the material 30 is discharged by the force of gravity via an output port which terminates in an output flange 96.

In this embodiment the input flange 94 is round, as illustrated in Figure 17. As previously discussed, the output flange 96 is

rectangular, as illustrated in Figure 18. The rectangular output flange permits the inner walls of the housing to be aligned with the gap in the seat 92. This construction eliminates all of the ledges associated with other embodiments of the invention.

5 Stated another way, all spaces interior to the housing 84 are open to freely drain under the force of gravity, through the discharge port.

Alternate embodiments of the rotor are illustrated in Figures 19 and 20. In each of these embodiments the rotor comprised of two

10 trunnions, a cylinder and two trunnion filler portions. Both of these features are discussed in more detail below.

Each of the trunnions is comprised of a disc 101, a ring 102, a shaft 106 and a filler portion 102. More specifically, the disc 101 may be formed of sheet metal, such as stainless steel, by

15 cutting the outer diameter with a conventional laser cutter. Similarly, the hole may be formed by laser cutting followed by reaming. The ring 102 may be formed either from tubing or from rolled sheet metal with the ends joined by welding. The disc 101 may be joined to the ring 102 by welding with the outer surface

20 of the trunnion machined to the desired diameter.

The shaft 106 is formed from suitable bar stock by conventional machining. The shaft 106 can be conveniently attached to the disc 101 by welding after the shaft 106 has been inserted into the disc with an interference fit. Conventional welding

25 techniques may be used to weld the shaft to 106 the disc 101 along the outer diameter of the shaft 106.

A trunnion filler portion 104 is positioned inside the ring portion 102 and adjacent the disc 101 to prevent a ledge from being formed in an area where the ring 102 joins the disc 101.

30 Such a ledge could accumulate portions of the material being transferred through the valve.

The cylinder 108 has two trunnions positioned in the ends

thereof.

The cylinder 108 may be formed of tubing or rolled sheet material. In either case the cylinder 108 may be attached to the trunnion by plug welding the cylinder 108 to the outer surface
5 of the ring 102.

Rotor embodiments illustrated in Figures 19 and 20 differ in that the trunnions are reversed. Each of these embodiments has advantages, depending on the material to be transferred. For example, the embodiment illustrated in Figure 19 may have better
10 draining characteristics in that the edge of the trunnion form a straight wall coincident with the edge of the opening in the rotor. By contrast, the embodiment illustrated in Figure 10 has more volume. This permits more material to be transferred with each revolution of the rotor.

15 Figures 21, 22 and 23 respectively illustrate the top, cross section and the end view of the seat 42. From the top view it is clear the seat includes a circular opening permitting the seat retainer 92 to have a cylindrical cross section. Flutes extend longitudinally across the surface of the seat 42. These flutes
20 function in a manner that was previously described with respect to the other embodiments.

Other salient features of the seat 93 are illustrated in Figures 22 and 23. This gap spacing serves in the same manner previously describe with reference to the first embodiment.

25 The compression seal 92 for use with this embodiment of the invention is illustrated in Figure 24. This compression seal can be made of silicon foam and is similar to the compression seals previously described. The principle difference between this and the previously described compression seals is that the dimensions
30 have been adjusted to conform the changed dimensions of the rotor and the seat.

The application of the material control adapter to a typical

vaneless rotor air lock valve is illustrated in cross section in Figure 25. As is conventional, the illustrated vaneless rotor air lock valve includes a housing 140 adapted to support a rotor 142 such that the rotor can be rotated about its longitudinal axis A. A suitable rotor seal 144 contacts the outside surface of the rotor 140 to seal the input port 146 from the output port. Pressure is applied to the outer surface of the rotor seal by a compression seal 149. The input port and the output respectively terminate in the input flange 150 and the output flange 152.

As previously discussed, the material control adapter 154 is positioned to contact and mate with the input flange 150. Suitable fasteners, such as bolts 156 and 158, may be used to secure the material control adapter 154 in the desired fixed relationship with the input flange 150. If desired, a suitable seal, not illustrated, such as a gasket, may be positioned between the input flange 150 and the material control adapter 154. As optional seal retainer 159 can be positioned as shown to hold seals 144 and 149 in place.

As can be seen from Figure 25, the material control adapter 154 includes a generally ring shaped top portion with a cone shaped side portion attached to its inner edge. A substantially constant distance is provided between the bottom end of the material control adapter and the rotor by a bottom end in the form of an intersection cone and a cylinder.

With prior art designs, when an attempt was made to transfer materials such as plastic pellets often used in the plastics industry through the vaneless rotor air lock valve, some of the pellets would become trapped between the edge of the opening in the rotor and the edge of the rotor seal. In these designs the most practical material for the rotor seal was some form of plastic. As the pellets were squeezed between the rotor edge and the rotor seal, at closure point 160, the edge of the rotor seal was damaged. Additionally, some or all of the pellet could be forced between the outer surface of the rotor and the inner

surface of the rotor seal. The net result was rapid deterioration of the rotor seal.

As illustrated in Figure 25, the material control adapter provides a predetermined space 162 between the bottom edge of the material control adapter and the edge of the rotor seal. This feature is believed to be responsible for directing materials flowing through the vaneless rotor air lock valve away from the intersection of the edges of the rotor seal and the opening in the rotor, thus essentially eliminating the problem with pellets being trapped in this region.

In Figure 25, the input port and the output port are illustrated as round. This embodiment is not a limitation of the shape of the input port and the output port. As will be appreciated by those skilled in the art the material control adapter 154 could be formed with a rectangular upper surface and applied to vaneless rotor airlock valves having rectangular input ports.

Claims:

1. Material transfer apparatus including a seat for use in a vaneless rotor airlock valve, said seat comprising in combination:
a curved member including a first opening therein for mating with apparatus adapted to direct material from the input port of a vaneless rotor airlock valve to the interior of a rotor comprising a portion of a vaneless rotor airlock valve, an interior surface for mating with the exterior surface of said rotor of said vaneless rotor airlock valve to maintain a pressure seal between the input and output ports of said vaneless rotor airlock valve, first and second ends adapted to maintain a selected distance therebetween during normal operation of said vaneless rotor airlock valve, an outer surface adapted to respond to a pressure applied thereto to maintain said interior surface in sealing contact with said rotor.
2. A seat for use in a vaneless rotor airlock valve comprising a curved member having a radius and a first opening therein through which material to be transferred flows to the interior of a rotor comprising a portion of a vaneless rotor airlock valve, a second opening through which material to be transferred flows from the interior of said rotor to the output port of said vaneless rotor airlock valve, an outer surface adapted to respond to pressure applied thereto to adjust the curvature of the interior surface of said seat to maintain a pressure seal between said seat and said rotor comprising a portion of said vaneless rotor airlock valve.
3. A seat in accordance with Claim 2, said seat further including first and second ends, the distance between said first and said second ends being selected such that for all expected operating conditions of said valve said distance is maintained greater than
a selected value.

4. A vaneless rotor airlock valve comprising in combination:
- a) a housing including first and second intersecting bores, opposite ends of said first bore comprising input and output ports;
 - b) a rotor including an outer surface having an opening therein to form a cavity in said rotor;
 - c) bearing means for supporting said rotor in said second bore;
 - d) a split cylindrical sealing structure positioned between said outer surface of said rotor and the inner surface of said second bore; and
 - e) pressure means of applying pressure to the outer surface of said sealing structure to maintain a pressure seal between the outer surface of said rotor and the inner surface of said sealing means to maintain pressure isolation between said input and said output ports.
5. Apparatus for controlling the flow of material into a rotor of a vaneless rotor air lock valve comprising a device having a bottom end adapted to mate with the rotor of said vaneless rotor air lock valve, an upper end having a surface adapted to mate with the input flange of said vaneless rotor air lock valve, and a side portion extending between said bottom end and upper ends to form a passageway for flow of material transferred through said vaneless rotor airlock valve with the dimensions of said bottom end being selected to provide a selected distance between the outer surface of said bottom end and the edge of the entry way in the rotor seal of said vaneless rotor airlock valve.

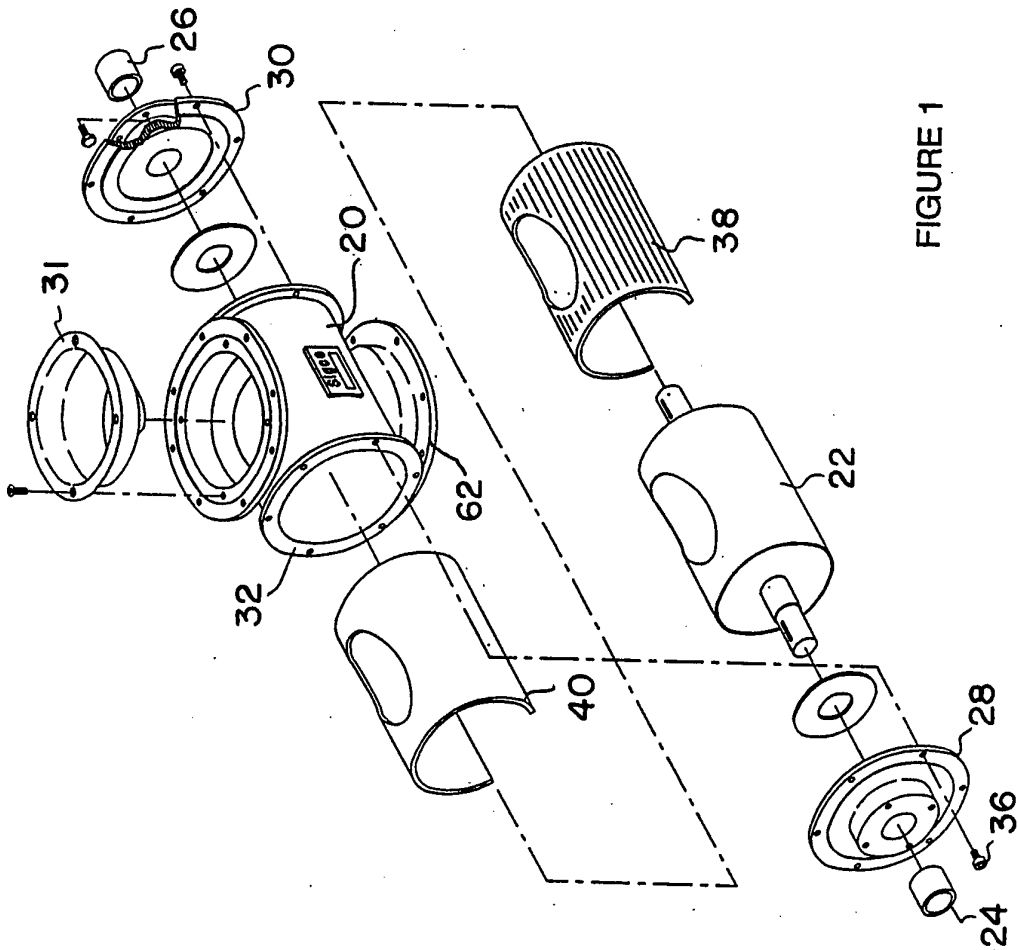


FIGURE 1

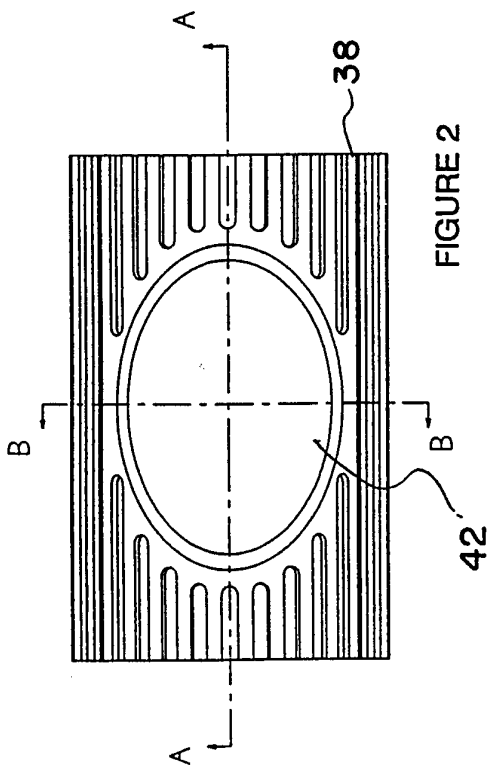


FIGURE 2

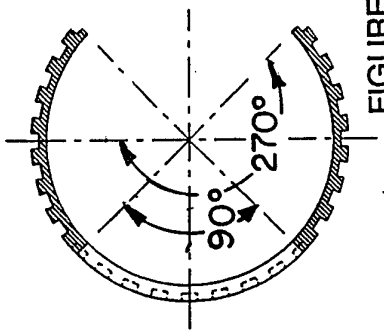


FIGURE 4

SECTION B-B

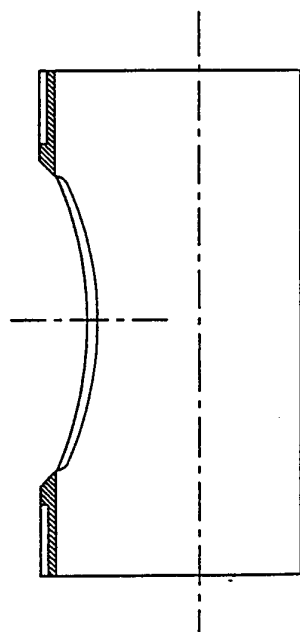


FIGURE 3

SECTION A-A

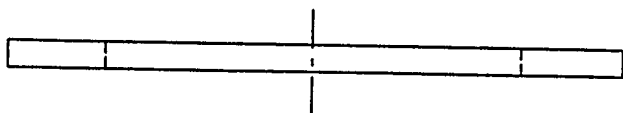


FIGURE 4B

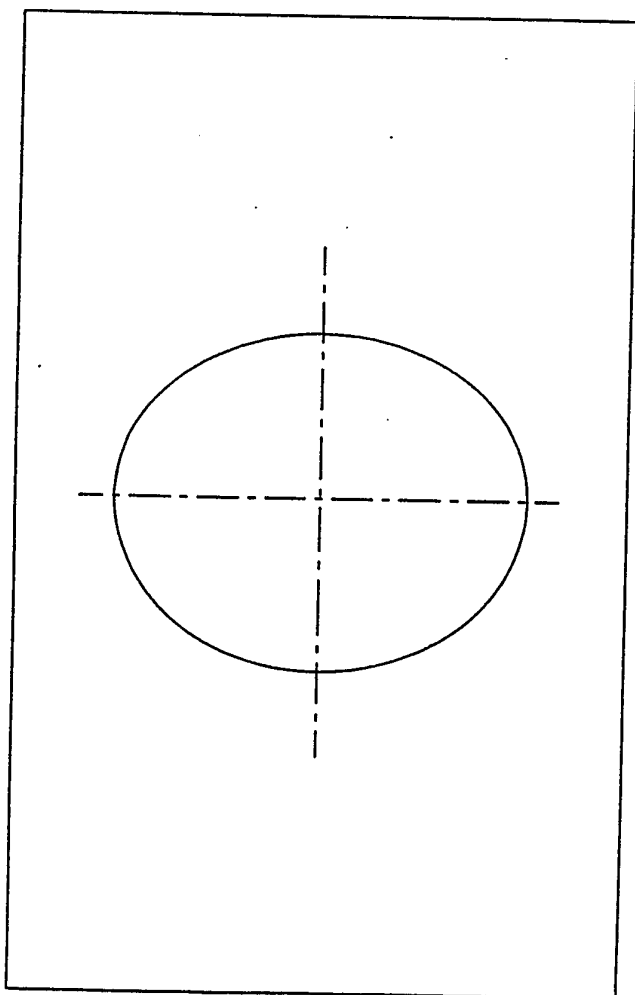


FIGURE 4A

COMPRESSION SEAL

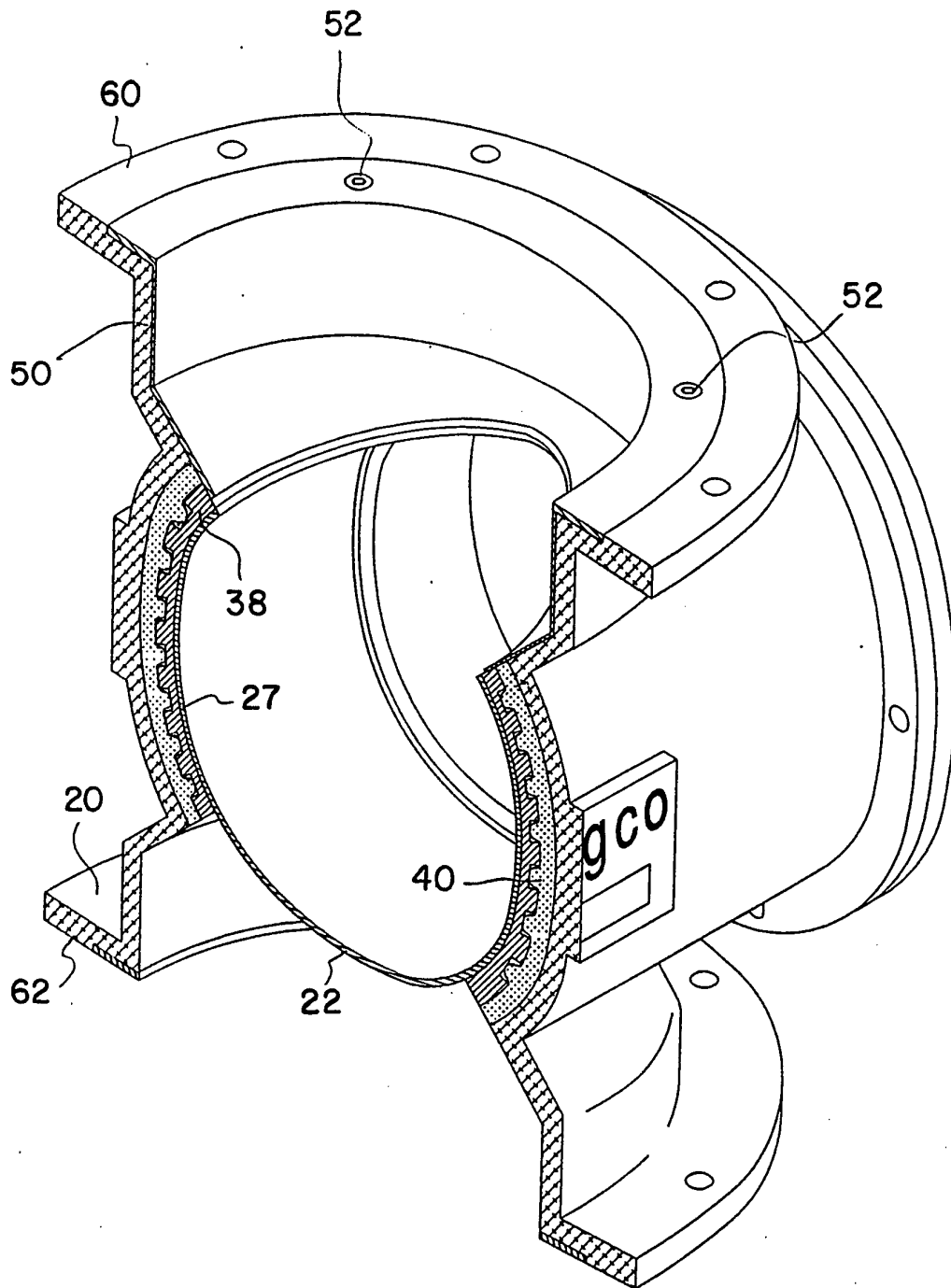


FIGURE 5

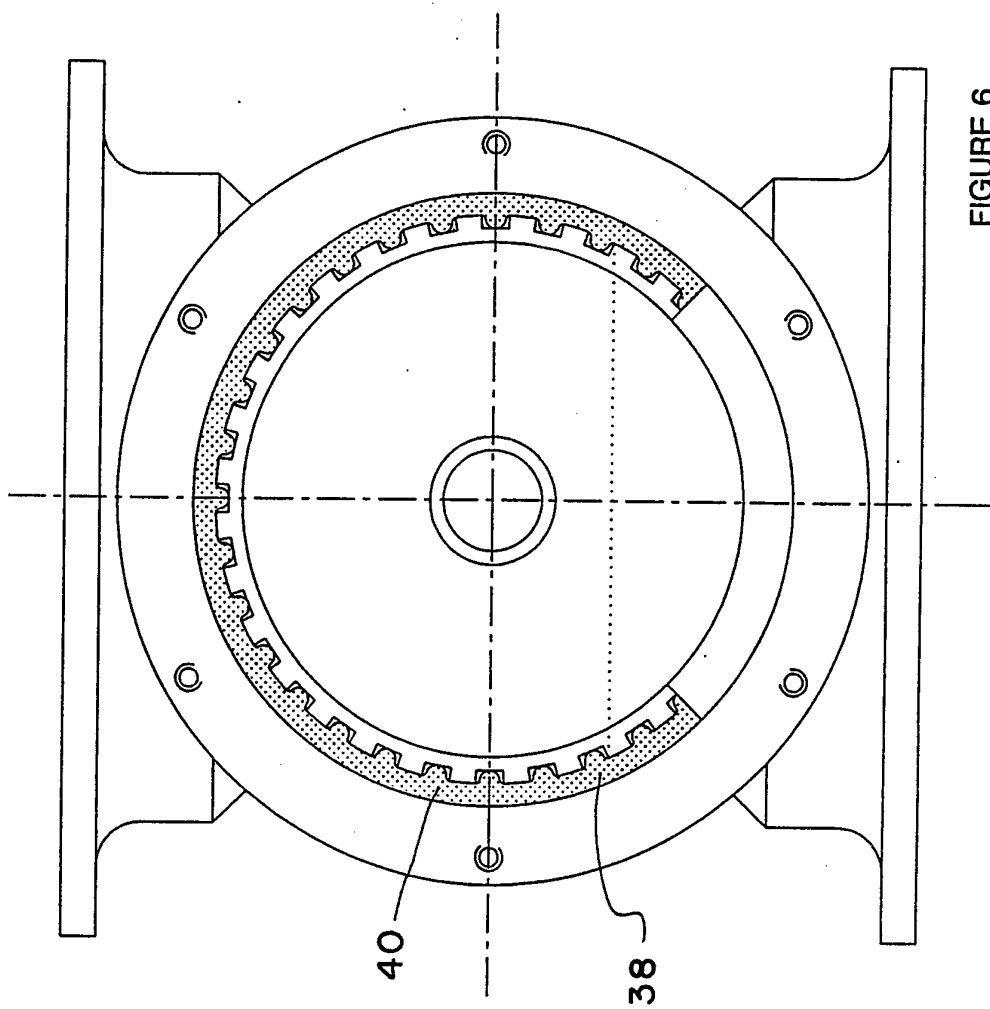


FIGURE 6

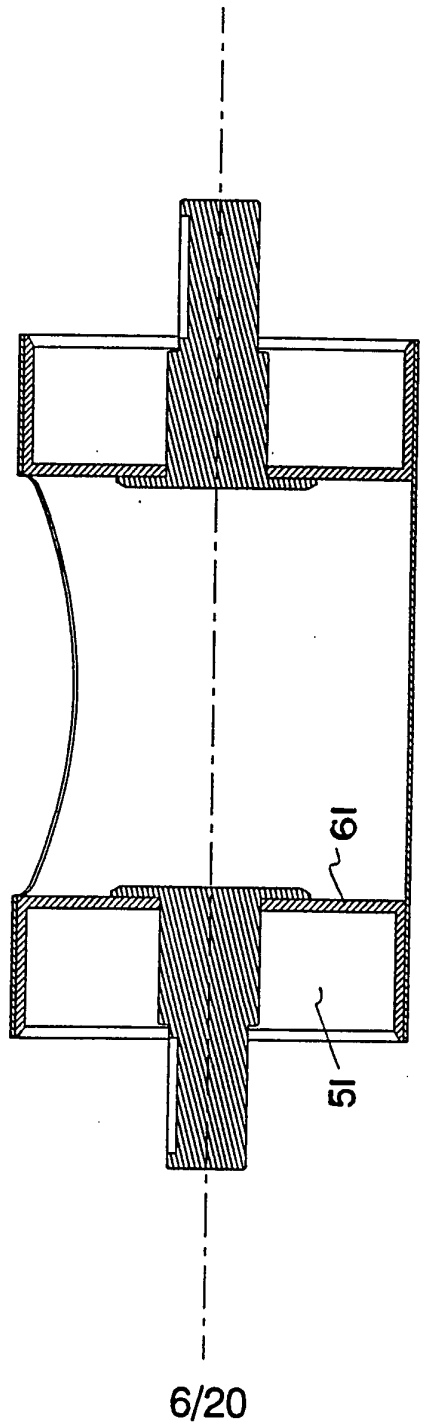


FIGURE 7

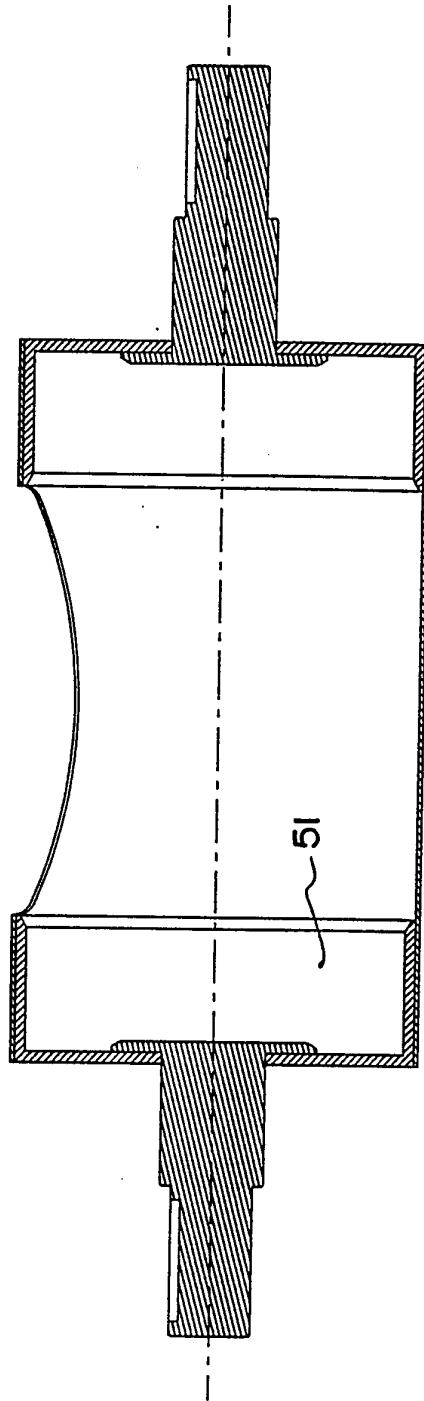


FIGURE 8

7/20

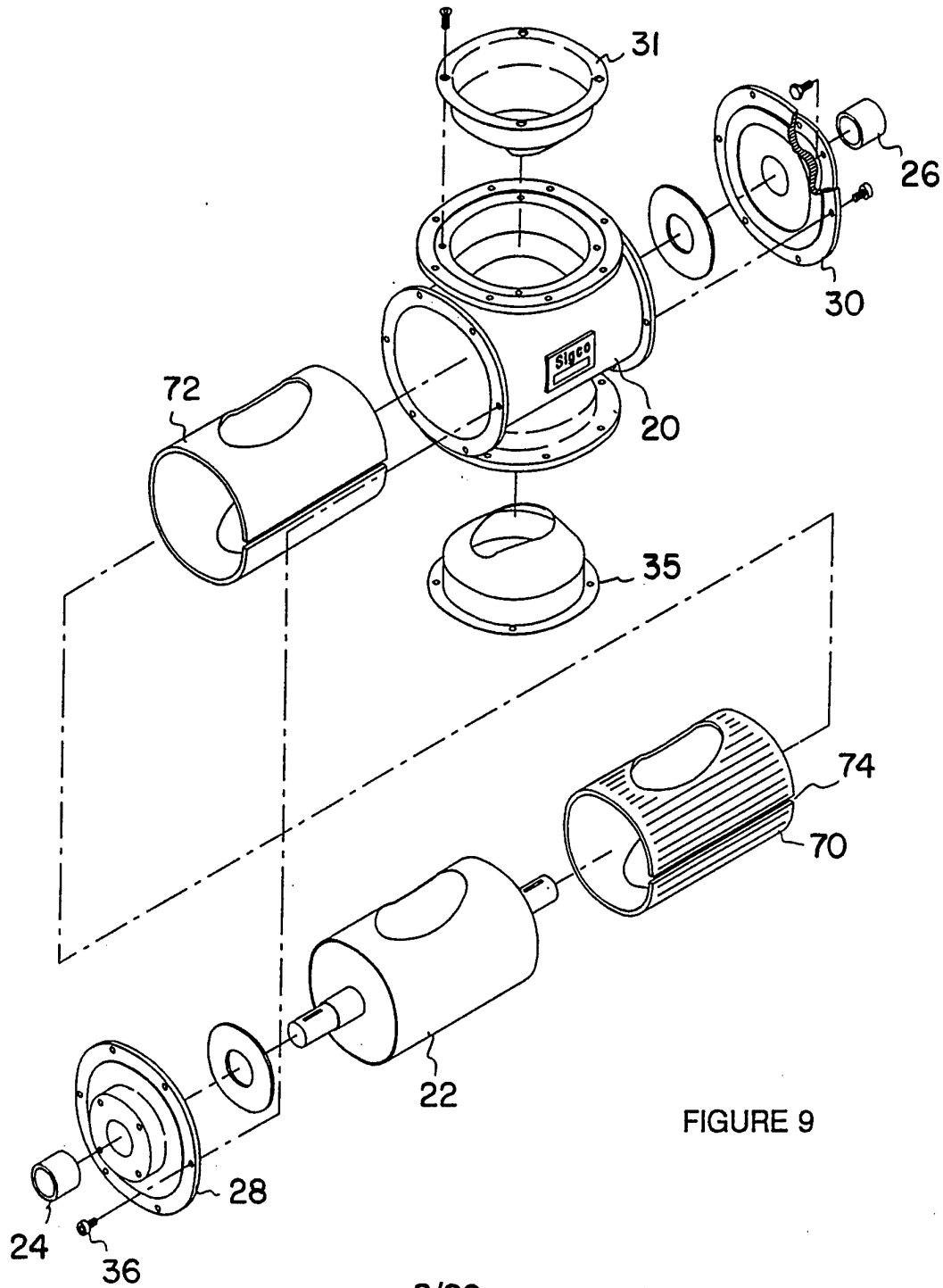


FIGURE 9

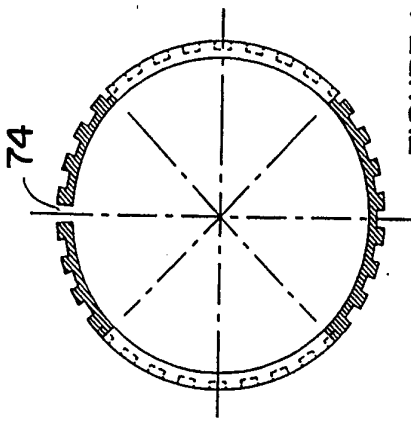


FIGURE 12

SECTION B-B

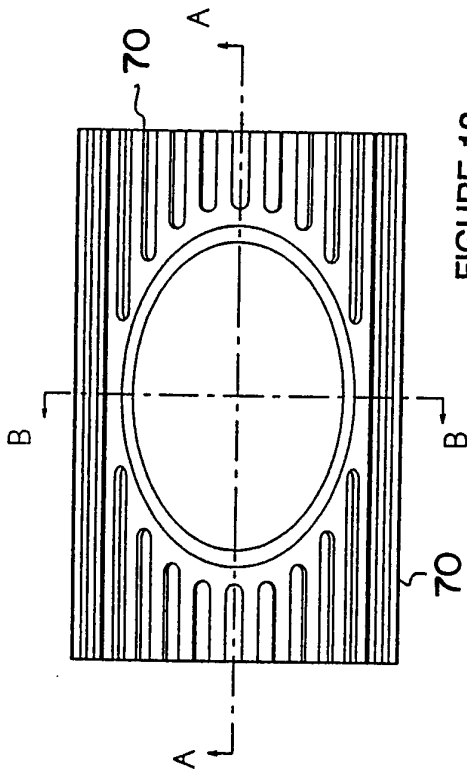


FIGURE 10

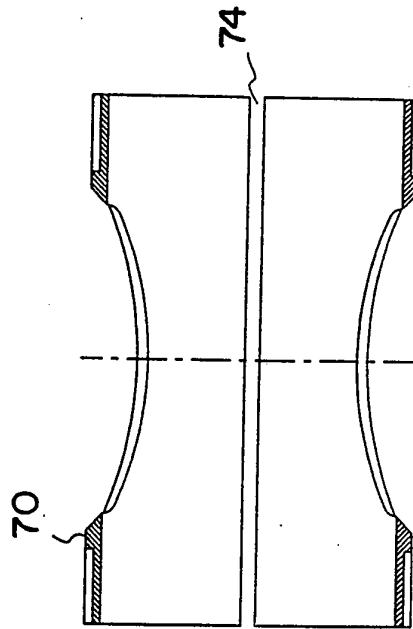


FIGURE 11

SECTION A-A

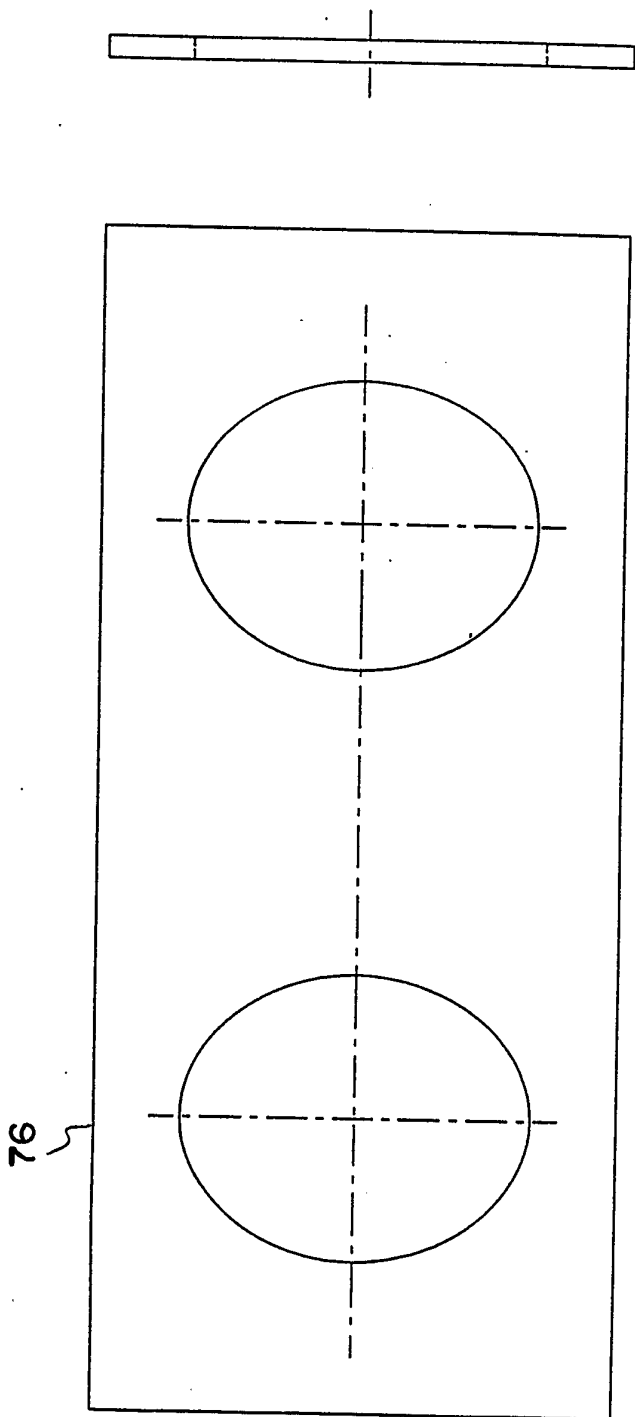


FIGURE 13A

FIGURE 13
COMPRESSION SEAL

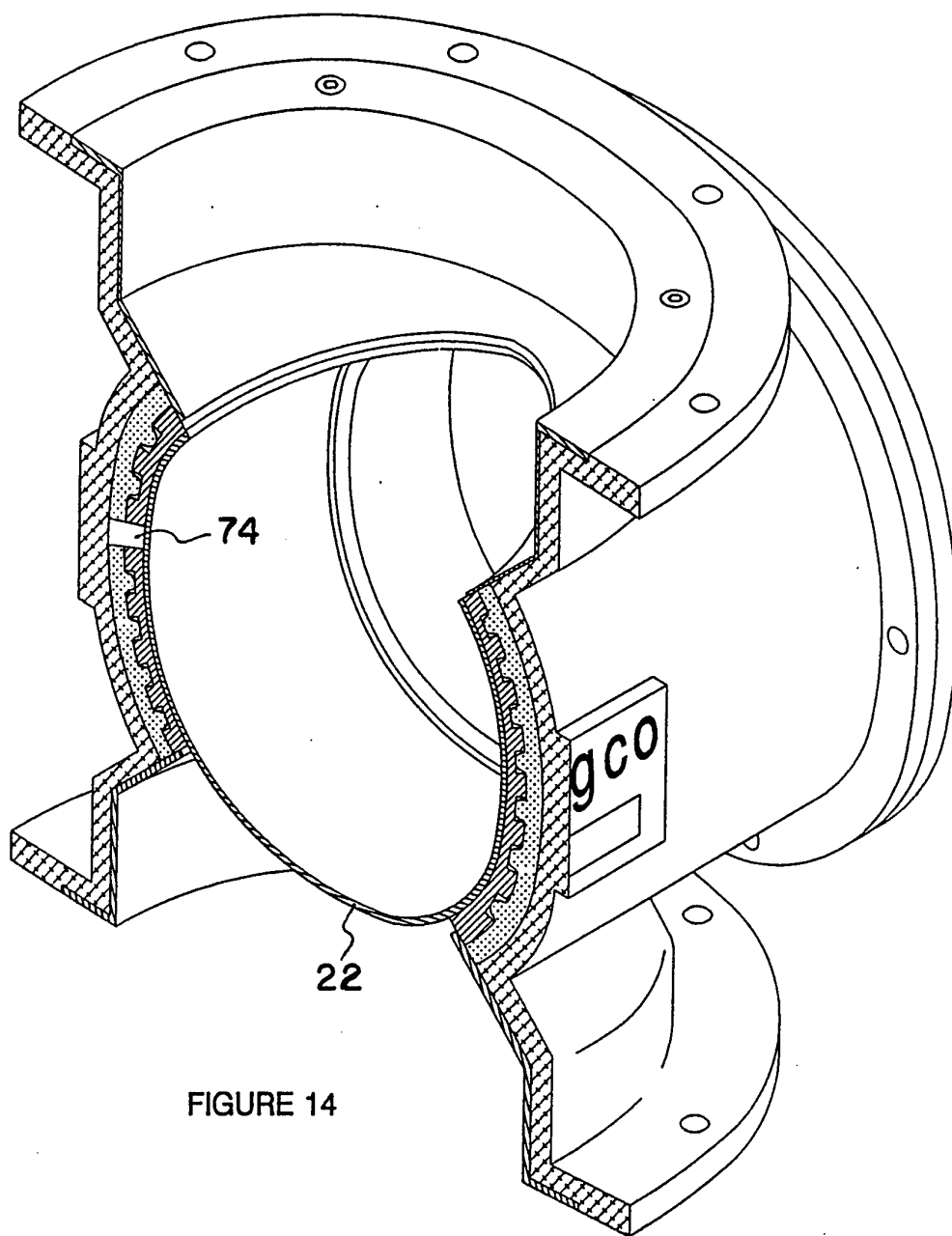


FIGURE 14

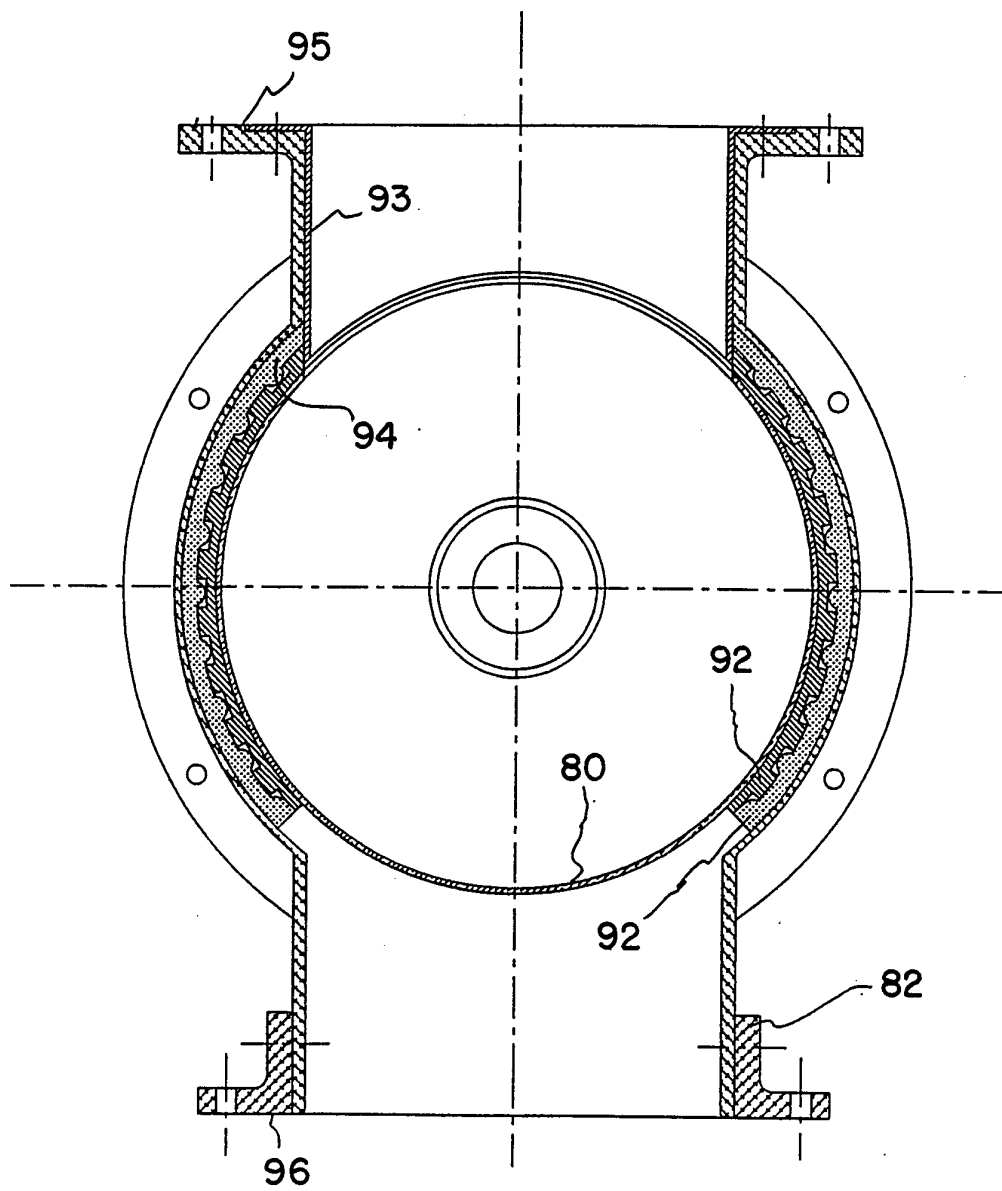


FIGURE 15

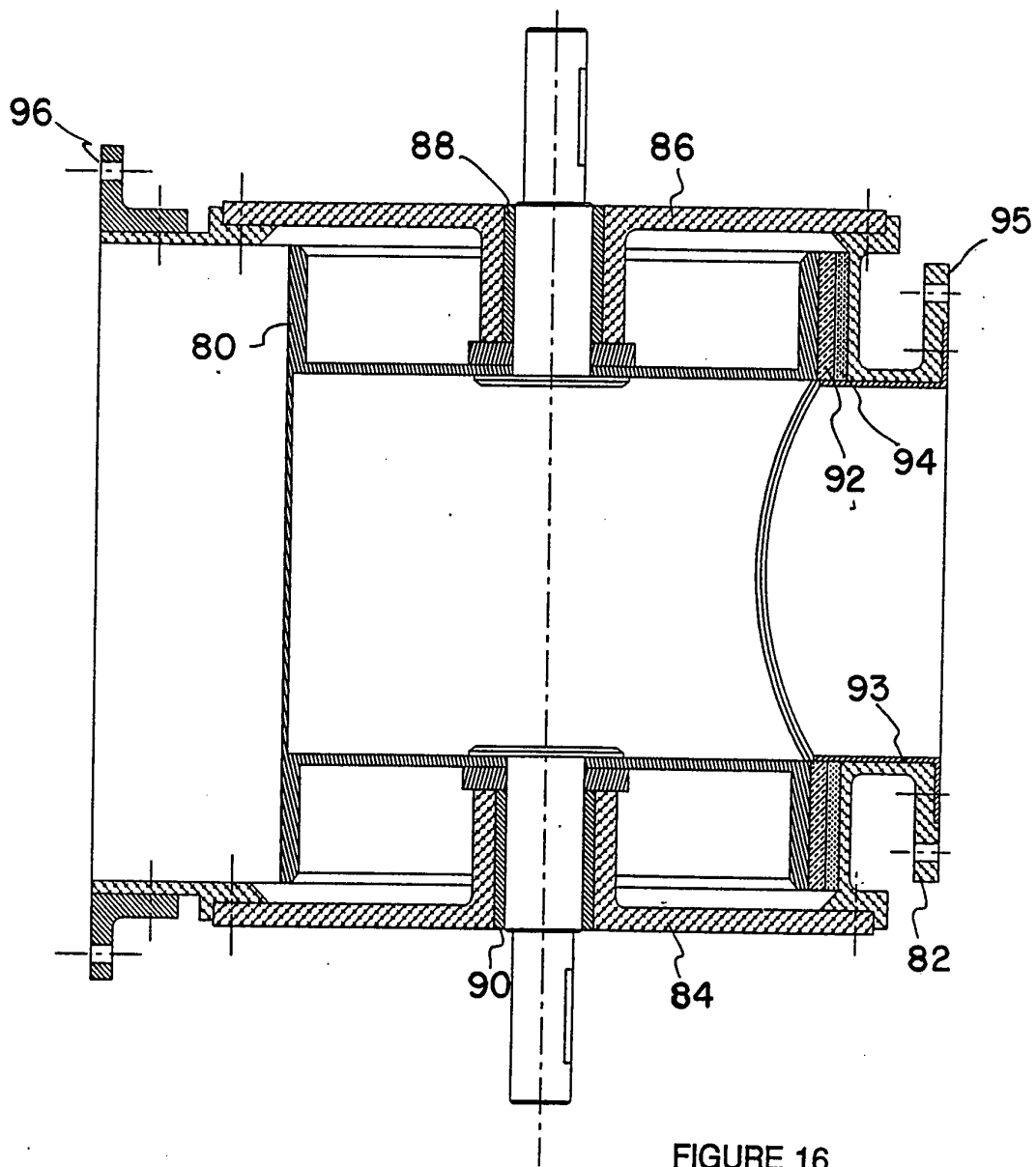


FIGURE 16

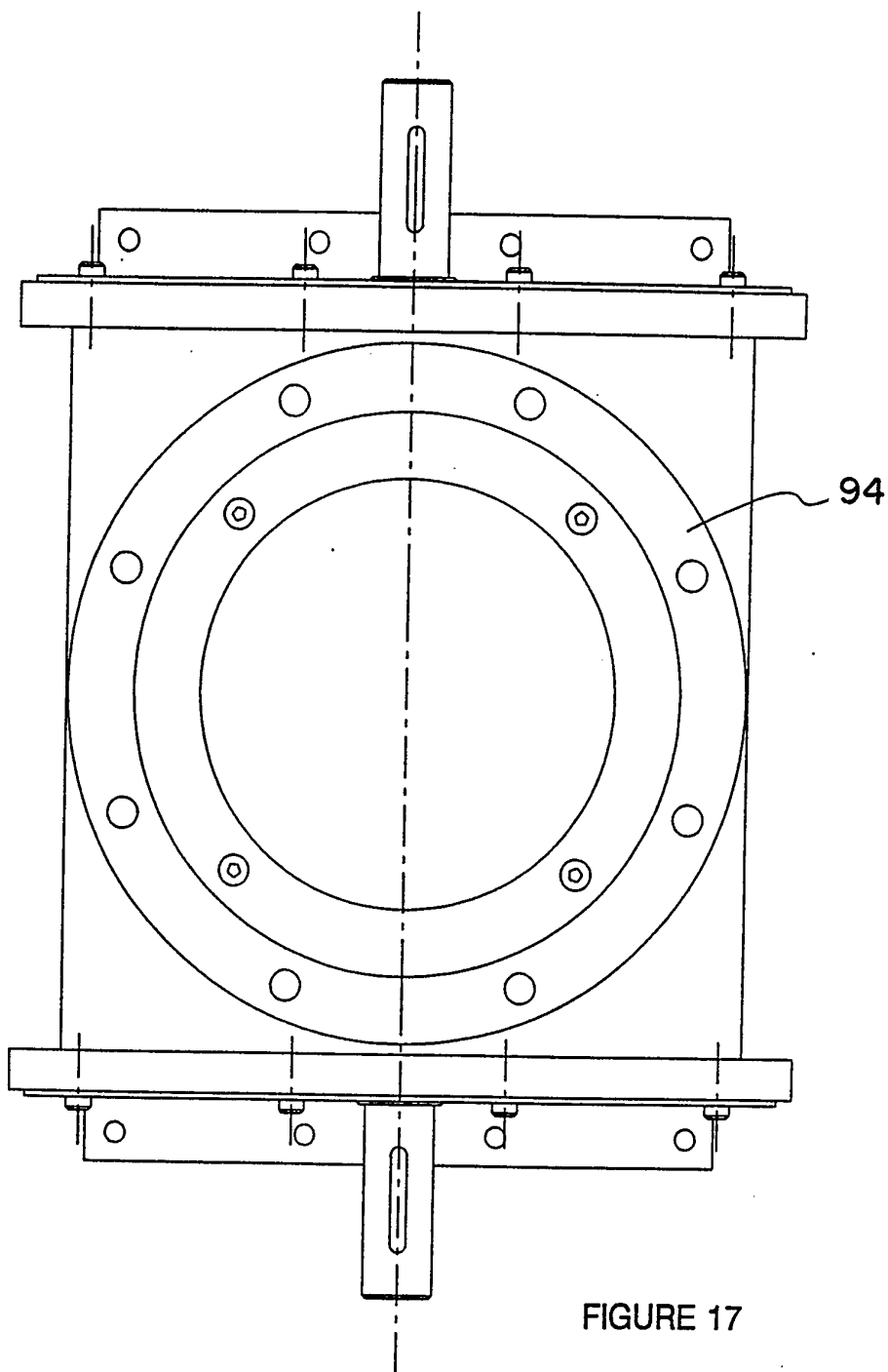


FIGURE 17

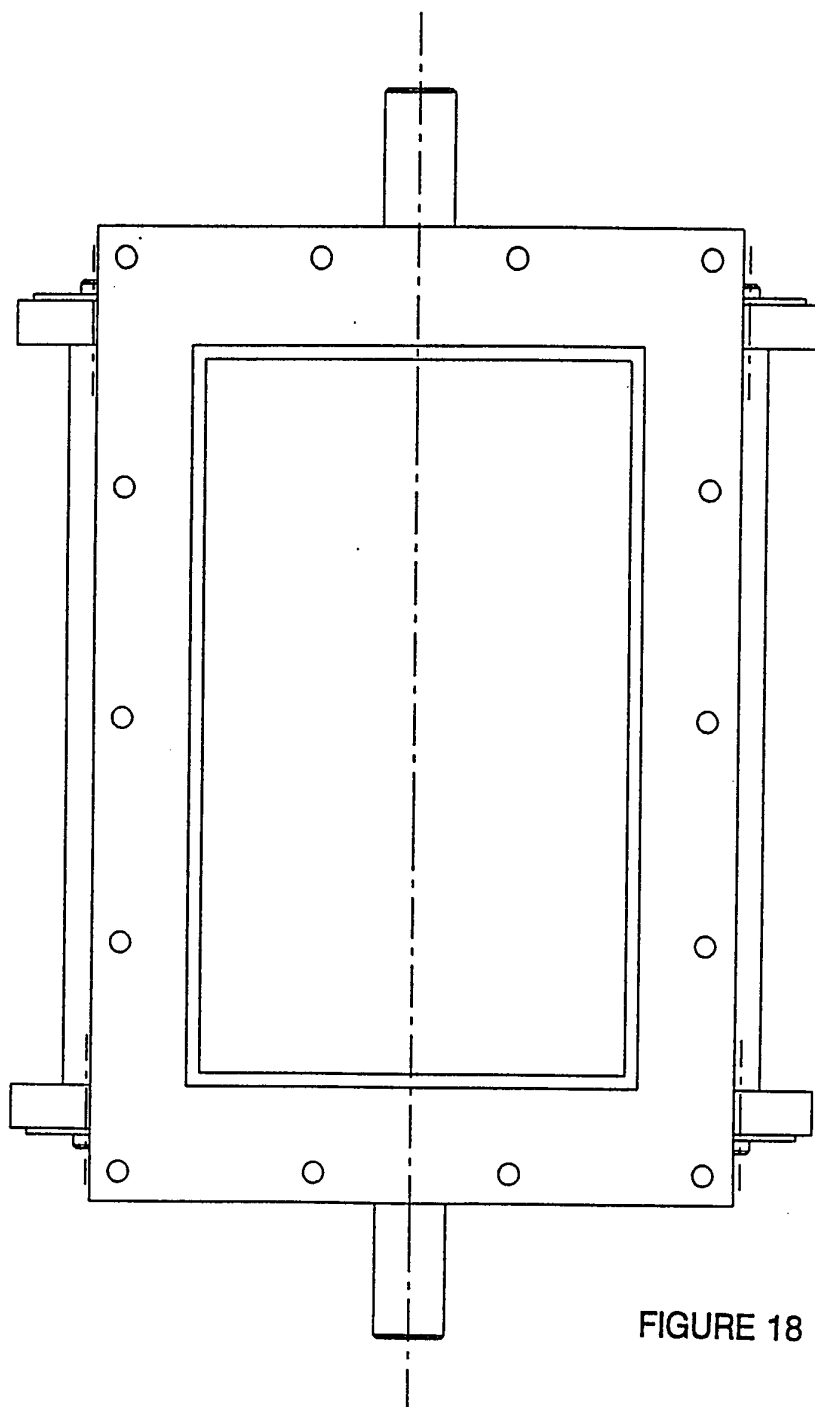


FIGURE 18

15/20

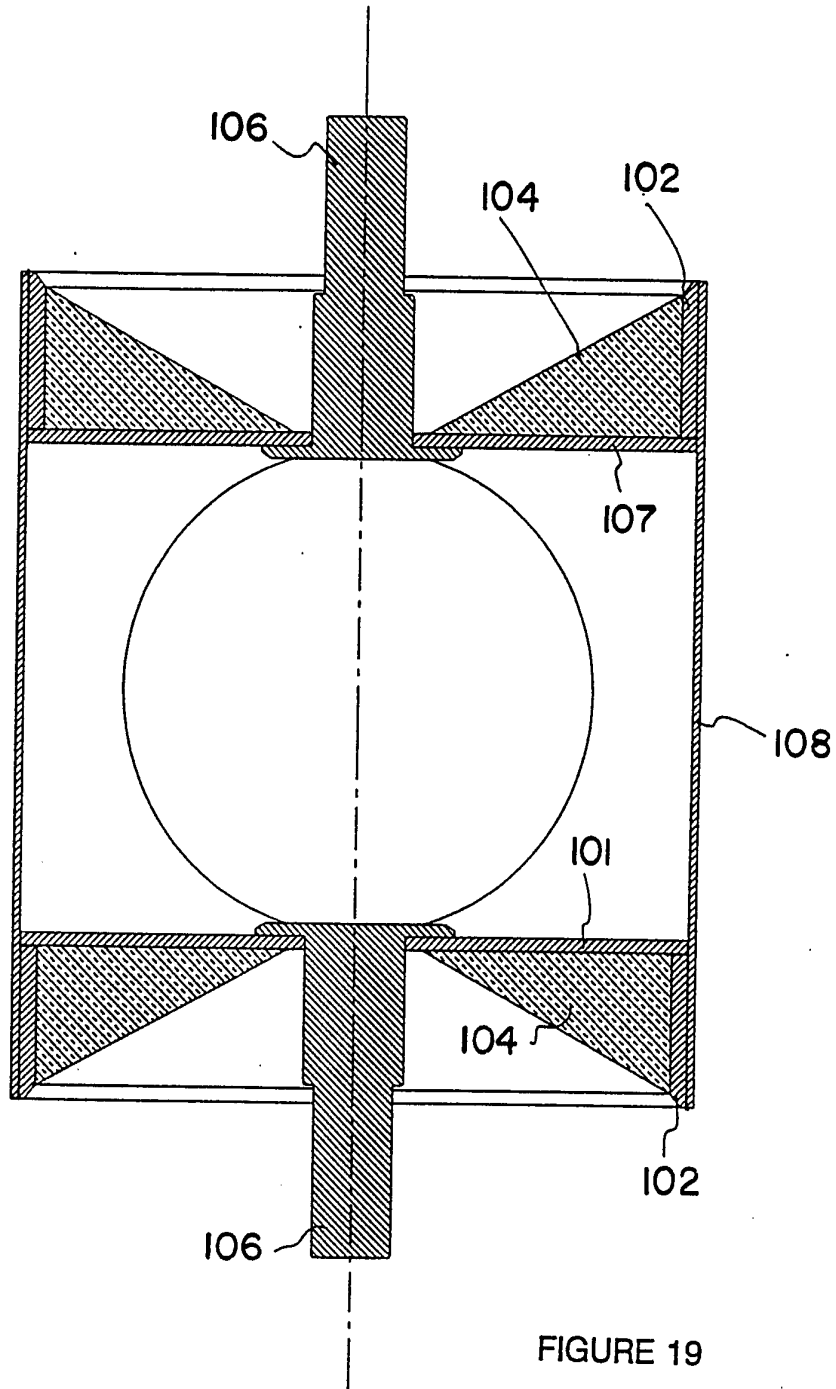


FIGURE 19

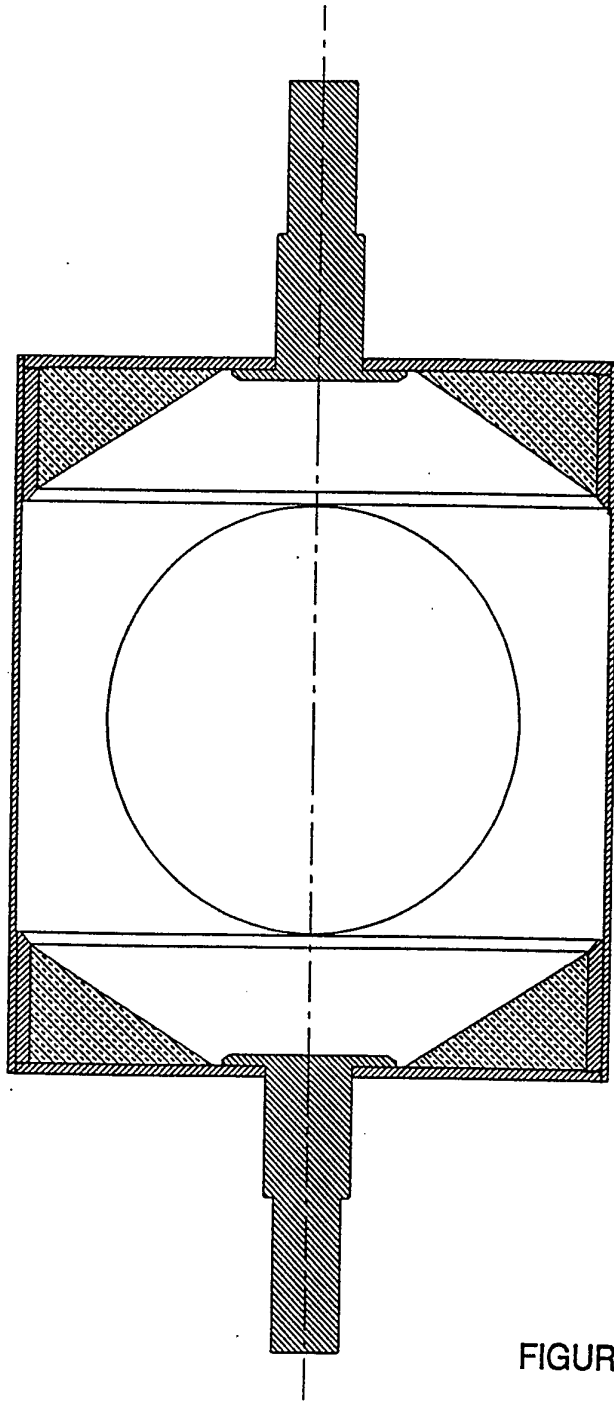


FIGURE 20

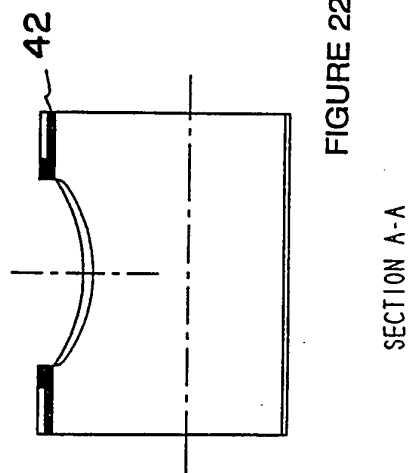
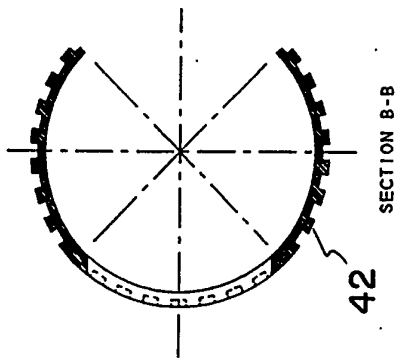
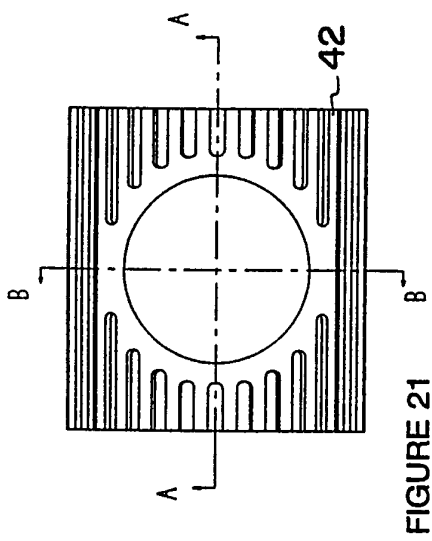




FIGURE 24A

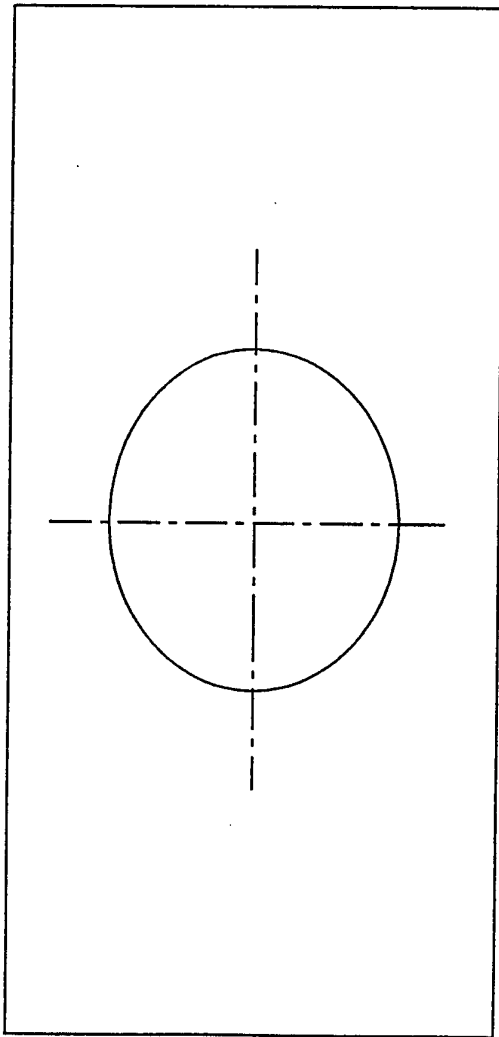


FIGURE 24

COMPRESSION SEAL

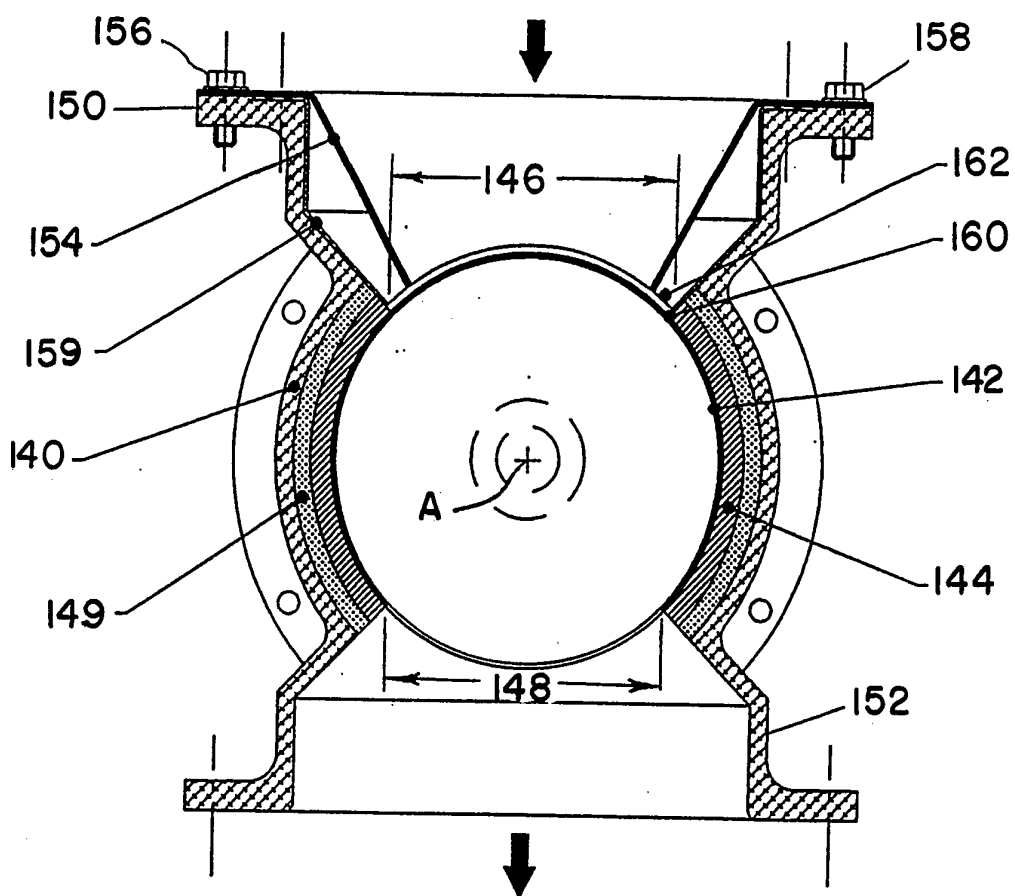


FIGURE 25

INTERNATIONAL SEARCH REPORT

International application No.
PCT/US94/12414

A. CLASSIFICATION OF SUBJECT MATTER IPC(6) :G01F 11/10 US CL :222/368 According to International Patent Classification (IPC) or to both national classification and IPC		
B. FIELDS SEARCHED Minimum documentation searched (classification system followed by classification symbols) U.S. : 222/363, 368, 444,452; 251/309,314 Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)		
C. DOCUMENTS CONSIDERED TO BE RELEVANT		
Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
Y	US, A, 5,029,517 (SIGMON) 09 July 1991, see the entire document.	1-3, 5
Y	US, A, 4,316,559 (MCLEMORE) 23 February 1982, see the entire document.	1-3, 5
A	US, A, 4,684,040 (JONOVIC ET AL.) 04 August 1987	
A	US, A, 3,139,996 (WELTY) 07 July 1964	
A	US, A, 3,750,902 (STARRETT) 07 August 1973	
A	US, A, 2,907,499 (AGRONIN) 06 October 1959	
A	US, A, 3,370,335 (FREED) 27 February 1968	
<input type="checkbox"/> Further documents are listed in the continuation of Box C. <input type="checkbox"/> See patent family annex.		
* Special categories of cited documents:	"T"	later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention
"A" document defining the general state of the art which is not considered to be of particular relevance	"X"	document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone
"E" earlier document published on or after the international filing date	"Y"	document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art
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"O" document referring to an oral disclosure, use, exhibition or other means		
"P" document published prior to the international filing date but later than the priority date claimed		
Date of the actual completion of the international search	Date of mailing of the international search report	
02 DECEMBER 1994	DEC 14 1994	
Name and mailing address of the ISA/US Commissioner of Patents and Trademarks Box PCT Washington, D.C. 20231	Authorized officer <i>Kevin P. Shaver</i> KEVIN P. SHAVER	
Facsimile No. (703) 305-3230	Telephone No. (703) 308-1113	