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(54) **MULTI-TURN HYDRAULIC ACTUATOR**

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(58) **Field of Classification Search** **251/14, 251/129.03, 249.5, 65; 137/554**

See application file for complete search history.

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(57) **ABSTRACT**

There is provided a multi-turn hydraulic actuator used to open/close a passageway in ground and sea industries including general industrial machines, oil refineries and chemical plants and power generating plants, among others, and more particularly, a multi-turn hydraulic actuator comprising: a hydraulic motor and a decelerator, among others, whereby high torque is efficiently obtained by using low electric power, compared to a conventional electric actuator, and whereby an operation part and a drive part are freely attached to/detached from at a user's necessity.

9 Claims, 4 Drawing Sheets

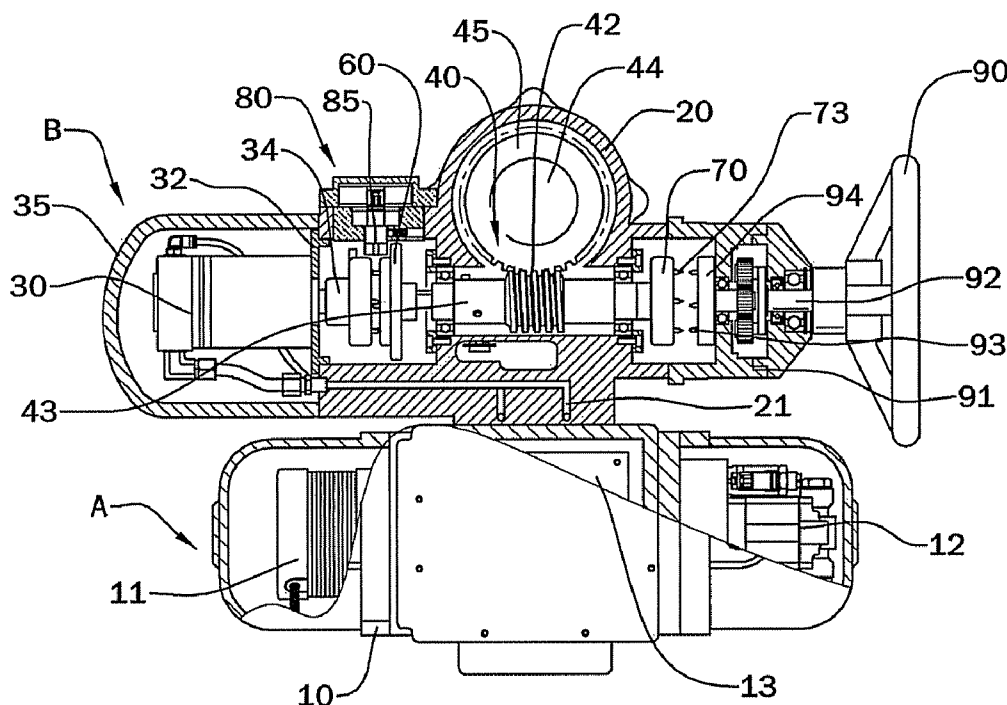


Fig. 1

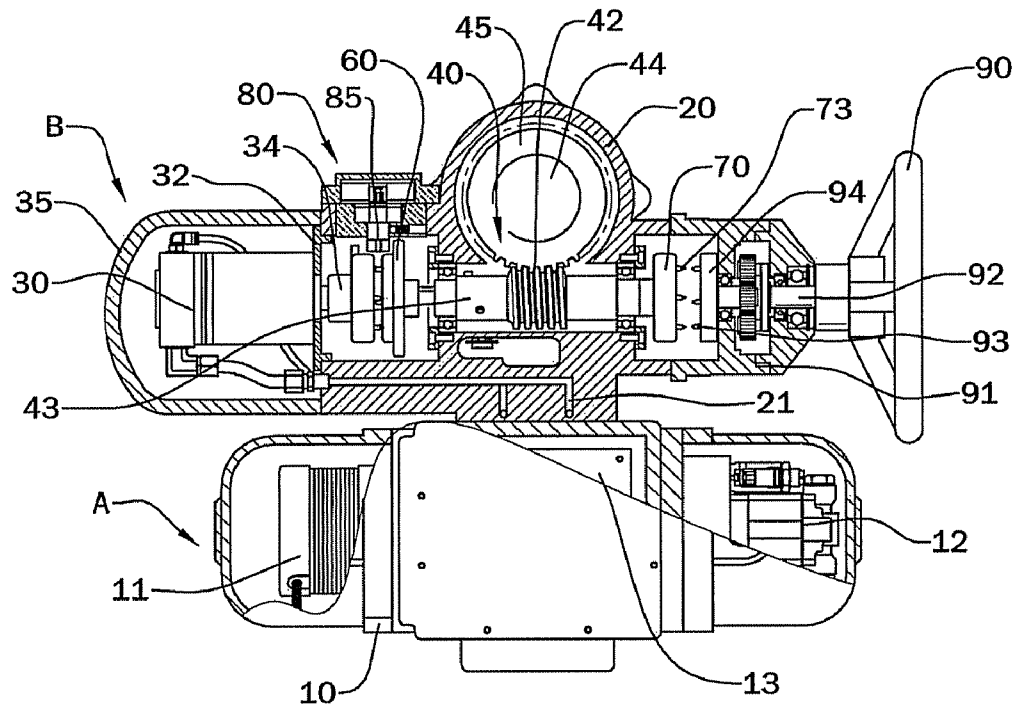


Fig. 2

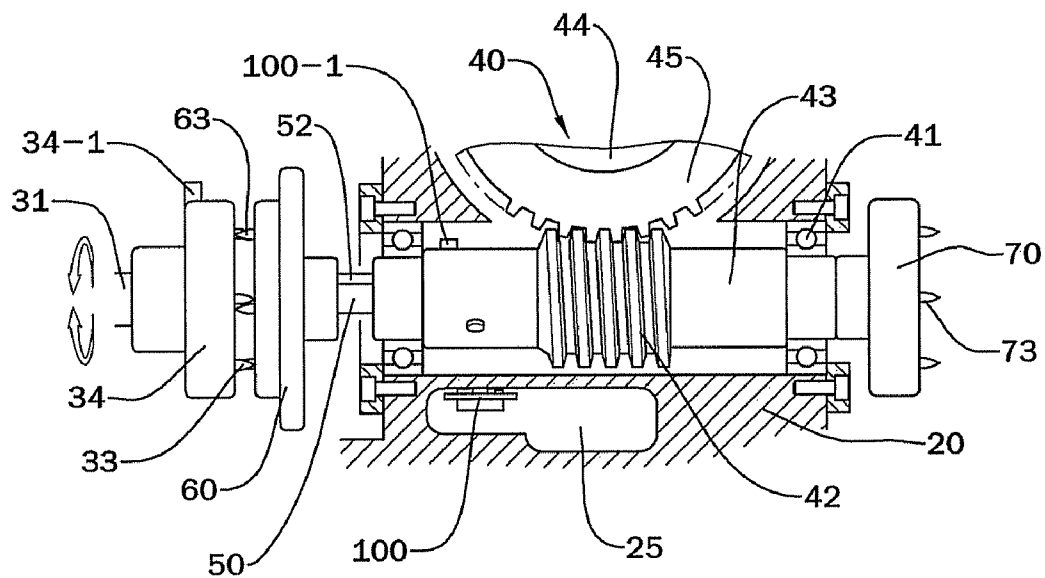


Fig. 3

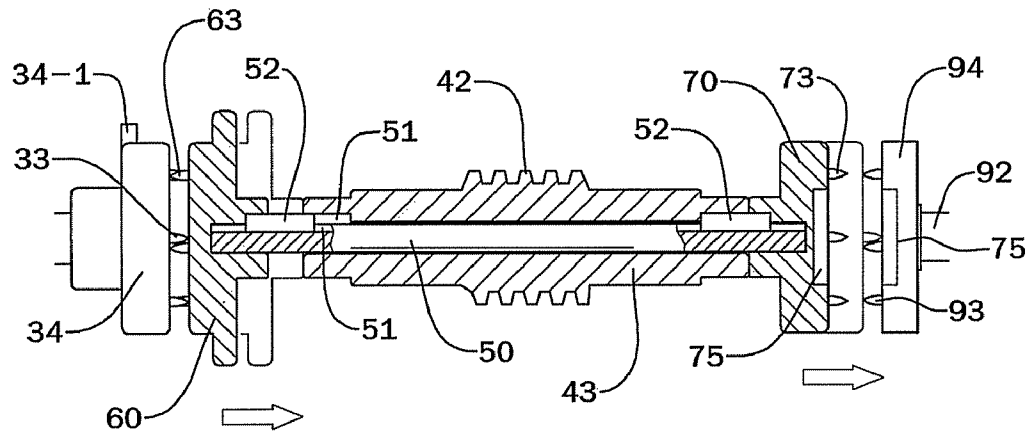


Fig. 4

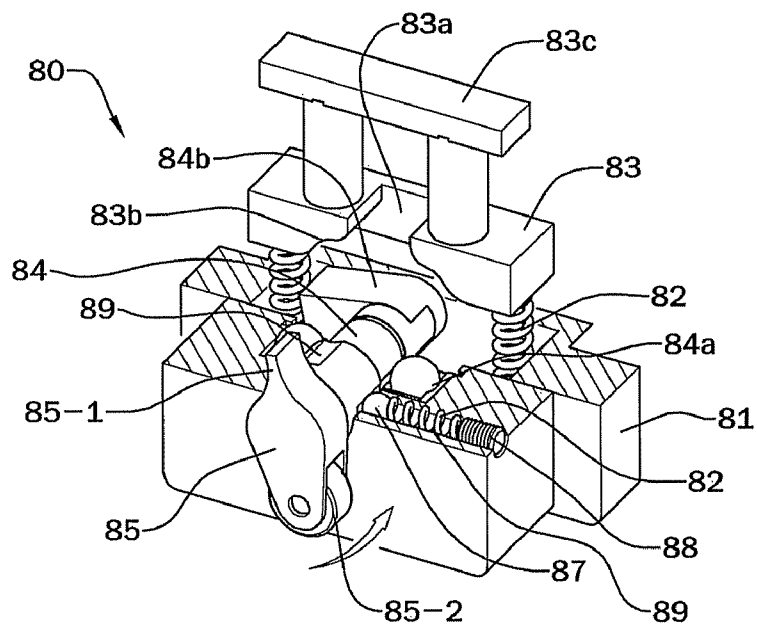


Fig. 5

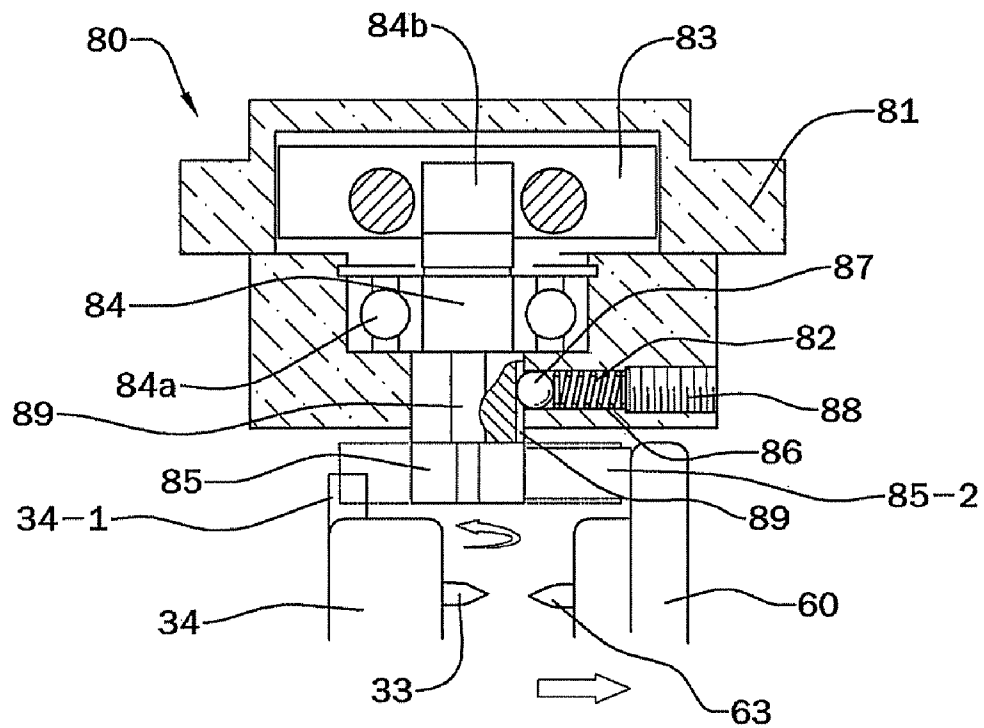


Fig. 6

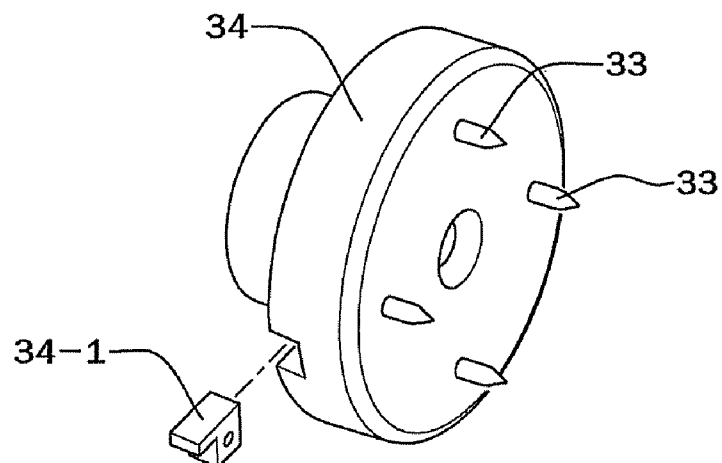


Fig. 7

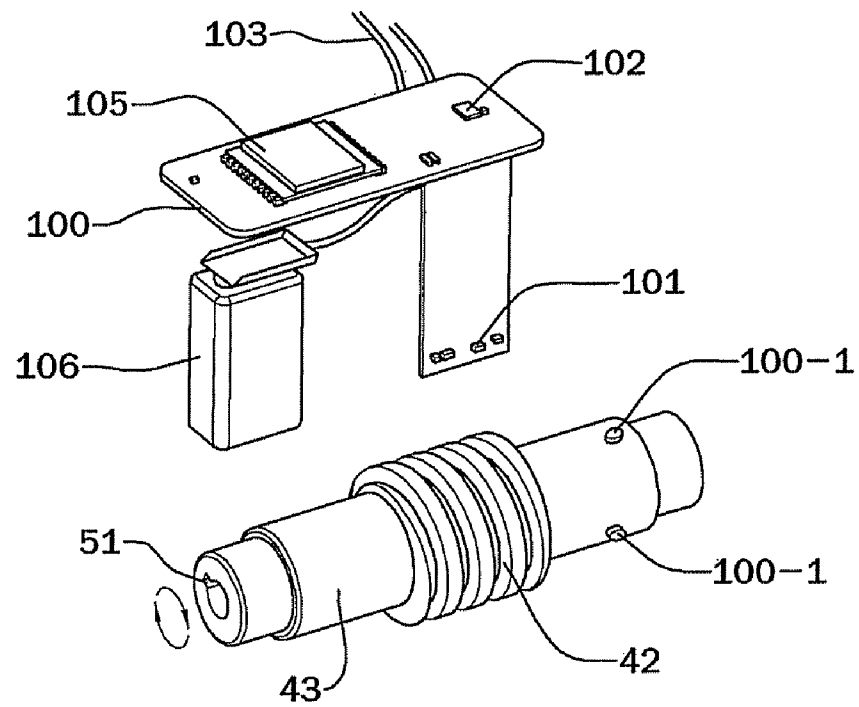
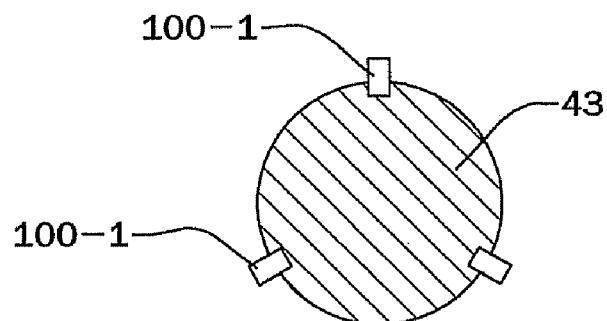


Fig. 8



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MULTI-TURN HYDRAULIC ACTUATOR**CROSS-REFERENCE TO RELATED PATENT APPLICATION**

This application claims the benefit of Korean Patent Application No. 10-2008-0095532, filed on Sep. 29, 2008, in the Korean Intellectual Property Office, the disclosure of which is incorporated herein in its entirety by reference.

BACKGROUND OF THE INVENTION**1. Field of the Invention**

The present invention relates to a multi-turn hydraulic actuator used to open/close a passageway in the ground and sea industries including general industrial machines, oil refineries and chemical plants, and power generating plants, among others, and more particularly, a technical field in which an actuator comprising a hydraulic motor and a decelerator is used to control the opening/closing of a valve which controls the flow of a fluid flowing through a path of a conduit, or a lock which is installed at a passageway.

2. Description of the Related Art

In general, major types of valve driving systems include a butterfly valve, a gate valve and a plug valve, among others, according to valve forms. Each valve is automatically driven by a 90 degree-turn or multi-turn actuator suitably selected according to use-environments. Examples of actuators include electric actuators, pneumatic actuators and hydraulic actuators by forms. Two functions, i.e., a stabilized actuation performance and a semi-permanent life span, are recognized as the most important factors in the actuators.

To control a valve installed in an industrial conduit or a lock installed in a passageway, a shaft of the valve or lock needs to be rotated, to perform the opening/closing operation of the valve or lock. The valve performs its operation generally by using an actuator which is controlled remotely or in the spot.

A conventional multi-turn electric actuator generally used comprises: a control unit including a 3-phase inverter converting single phase power into 3-phase power; a 3-phase electric motor operated by the control unit; a multi-unit with a rotation shaft installed perpendicularly to be rotated in either direction by the driving of the 3-phase electric motor; and a passageway switching unit for opening/closing a passageway.

In the aforementioned multi-turn electric actuator, when electric power is applied to the control unit including the 3-phase inverter, the 3-phase electric motor is driven. The motive power generated in the 3-phase electric motor is transmitted to a worm gear through a gear inside the multi-unit, thereby ultimately driving a drive block of the passageway switching unit installed at an end of the rotation shaft. Consequently, the opening/closing of the valve is controlled by the torque.

Further, the conventional multi-turn electric actuator is structured to monitor the operation of the gear inside the multi-unit, to perceive the extent of opening the valve and the drive of the actuator. For this purpose, an analog detection method and a digital detection method are used. In the former, an additional shaft is directly connected to the gear, to detect the number of times of rotation of the shaft by using variable resistance. In the latter, the number of times of rotation of the shaft is detected by using a sensor.

In a manual method of manually operating the conventional multi-turn electric actuator, a manual switch lever is rotated by human power, to apply a change to a combination

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of the gear inside the multi-unit, thereby changing an automatic mode to a manual mode.

When using the manual switch lever to change the mode of the actuator to the manual mode, sufficient rotational radius and space are required to operate the manual switch lever. Moreover, since all of the 3-phase electric motor, control unit, multi-unit, passageway switching unit and terminal unit forming the multi-turn electric actuator are structured in a single unit, it is impossible to install the manual switch lever in such a small space.

Further, in the analog detection method to perceive the extent of opening the valve and the drive of the actuator, an assembling process is very complicated since many gears are combined. Moreover, since the gear combination needs to be mechanically changed in order to set or correct a display of the extent of opening, it is necessary to open a cover. Therefore, this is regarded as a very annoying factor.

Further, in the digital detection method, one central processing unit of high-capacity is included in the control unit, to be used for the control and communication of the actuator. In addition, the central processing unit is used for a sensor of detecting the extent of opening. Therefore, high standby electric power is consumed at the moment when the actuator is not driven.

SUMMARY OF THE INVENTION

The present invention provides a multi-turn hydraulic actuator which solves the above problems of a conventional multi-turn electric actuator needing to be improved and has the new concepts making it easy to be installed and operated even in a limited space and making it possible to minimize the use of electric power.

It is therefore an object of the present invention to provide a multi-turn hydraulic actuator which uses a hydraulic motor, thereby using low electric power, significantly improving outputs and realizing fast response characteristics; which is formed in an assembly structure, thereby making it possible to detach/attach an operation part and a drive part so that only the drive part is installed in a limited space or in the spot where it is inconvenient to approach, and the operation part is separately installed at a desired place to be easily operated; which uses a manual mode switch to be input by a button, thereby minimizing a space for using the product; which uses a hydraulic pressure generator, thereby minimizing consumption of driving power; and which additionally includes a low-power processor needed for detecting the extent of opening, thereby minimizing standby electric power.

According to an aspect of the present invention, there is provided a multi-turn hydraulic actuator comprising: an operation part and a drive part being separately formed to be assembled/disassembled, wherein the operation part comprises: a hydraulic pressure generator including a motor and a pump being formed together in a single unit, and the drive part transmits a rotational force of a hydraulic motor to a decelerator and rotates a driving shaft forwardly/backwardly by the decelerated force.

The multi-turn hydraulic actuator further comprises: a manual mode switch and a manual handle, wherein the manual mode switch is installed between the hydraulic motor and the decelerator, to change the mode of the decelerator so as to be manually operated, and the manual handle is installed at the other end of the decelerator, to manually operate the decelerator.

The multi-turn hydraulic actuator further comprises: a non-touch digital detector for detecting the extent of opening, wherein the non-touch digital detector comprises: a number

of magnets for detecting the extent of opening, installed at a worm shaft forming the decelerator; and hole sensors installed to perceive signals of the magnets, thereby detecting a rotational direction of the worm shaft and the extent of opening.

BRIEF DESCRIPTION OF THE DRAWINGS

The above and other features and advantages of the present invention will become more apparent by describing in detail exemplary embodiments thereof with reference to the attached drawings in which:

FIG. 1 is a partial plan view illustrating a multi-turn hydraulic actuator according to an embodiment of the present invention;

FIG. 2 is an enlarged sectional plan view illustrating a motor disk, an automatic disk and a decelerator being assembled together in the multi-turn hydraulic actuator;

FIG. 3 is a horizontal sectional view illustrating an automatic disk, a moving shaft and a manual disk being moved when the mode of the multi-turn hydraulic actuator is changed to a manual mode;

FIG. 4 is a partial perspective view illustrating a manual mode switch in the multi-turn hydraulic actuator;

FIG. 5 is a partial horizontal sectional view illustrating the manual mode switch in operation;

FIG. 6 is an exploded perspective view illustrating a motor disk and a return protrusion in the multi-turn hydraulic actuator;

FIG. 7 is a separated perspective view illustrating a non-touch digital detector for detecting the extent of opening and a worm shaft in the multi-turn hydraulic actuator; and

FIG. 8 is a side sectional view illustrating magnets for detecting the extent of opening, which are installed at the worm shaft in the multi-turn hydraulic actuator.

[Brief description of reference numbers of major elements]	
A: operation part	B: drive part
10: hydraulic pressure generator	11: electric motor
20: housing of the drive part	25: detector receiving part
30: hydraulic motor	33, 63, 73, 93: rotation pins
34: motor disk	40: decelerator
43: worm shaft	44: driving shaft
45: worm wheel	50: moving shaft
60: automatic disk	70: manual disk
75: magnet	80: manual mode switch
82: elastic member	83: pusher
84: switch shaft	84b: pivot piece
85: release bar	87: fixing ball
89: ball receiving groove	90: manual handle
100: non-touch digital detector for detecting the extent of opening	
100-1: magnet for detecting the extent of opening	
101: hole sensor	102: low-power processor

DETAILED DESCRIPTION OF THE INVENTION

The present invention will now be described more fully hereinafter with reference to the accompanying drawings, in which preferred embodiments of the invention are shown.

As used herein, the singular forms “a”, “an” and “the” are intended to include the plural forms as well, unless the context clearly indicates otherwise.

As illustrated in FIG. 1, a multi-turn hydraulic actuator according to the present invention is characterized in that

hydraulic pressure generated by a hydraulic pressure generator 10 is supplied to a hydraulic motor 30 through a hydraulic channel 21 formed in a housing 20 of a drive part B or a separate hydraulic hose so that the hydraulic motor 30 is driven, a rotational force of the hydraulic motor 30 is transmitted to a worm shaft of a decelerator 40 directly connected to the hydraulic motor 30 so that a driving shaft 44 is rotated at a reduced speed, and the rotation of the worm shaft 43 is selectively manually operated by using a manual handle 90 and setting a manual mode switch 80.

An outline of the whole technical constitution of a multi-turn hydraulic actuator according to a preferred embodiment of the present invention to achieve the aforementioned characteristics will be described with reference to FIGS. 1 through 4. The multi-turn hydraulic actuator has the correlative constitution comprising: a hydraulic motor 30 positioned at a front end of a housing 20 of a drive part B and operatively rotated by hydraulic pressure generated by a hydraulic pressure generator 10; a decelerator 40 including a worm 42, a worm shaft 43, a driving shaft 44 and a worm wheel 45, wherein the worm shaft 43 with the worm 42 is rotatably installed in the middle of the housing 20 of the drive part B, and the worm wheel 45 connected to the driving shaft 44 in a single unit is engaged with the worm 42, transmitting a rotational force of the worm 42 to the driving shaft 44 at a reduced speed; a moving shaft 50 operatively connected to the worm shaft 43 so as to be rotatable by penetrating the worm shaft 43 and to selectively slide in either direction; an automatic disk 60 connected to one end of the moving shaft 50 and being operatively rotated by a number of rotation pins 63 selectively meeting a number of rotation pins 33 of a motor disk 34, respectively; a manual disk 70 connected to the other end of the moving shaft 50; a manual mode switch 80 wherein a release bar 85 positioned between the motor disk 34 and the automatic disk 60 pivots by operating a manual mode button 83c, so that the automatic disk 60 is moved so as not to interfere with the motor disk 34 and the worm shaft 43 is switched so as to be manually operated; and a manual handle 90 installed at the other end of the housing 20 of the drive part B, wherein rotation pins 93 of a handle disk 94 positioned at a front end of a handle shaft 92 selectively meet rotation pins 73 of the manual disk 70, respectively, thereby making the worm shaft 43 so as to be manually rotatable.

Below, the multi-turn hydraulic actuator having the constitution outlined above will be described, in more detail, to be easily carried out.

The hydraulic pressure generator 10 forming an operation part A in the multi-turn hydraulic actuator according to the present invention generates high hydraulic pressure. As illustrated in FIG. 1, a BLDC motor or electric motor 11 having high power efficiency is attached to one side of the hydraulic pressure generator 10. The hydraulic pressure is generated by a gear pump 12 directly connected to the electric motor 11. A controller 13 is provided at an upper position in the middle of the hydraulic pressure generator 10. The controller 13 has the functions of changing a speed of the hydraulic motor 30 and a direction of a hydraulic fluid and stopping driving of the electric motor 11, by controlling the operation of the hydraulic pressure generator 10.

The controller 13 does not need to be installed in the operation part A. When it is suitable to install the controller 13 in the drive part B by considering the conditions of the spot or the structural conditions of the operation part A and drive part B, the controller 13 may be installed in the drive part B.

The hydraulic pressure generator 10 is attachable to or detachable from the housing 20 of the drive part B directly connected to a valve installed at a conduit or a lock of a

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hydraulic channel. The hydraulic pressure generator **10** may be installed at the housing **20** of the drive part B in a single unit or it may be installed at a third place, separately from the housing **20** of the drive part B, without any restriction with respect to an installation space. Therefore, the hydraulic pressure generator **10** is highly applicable in use.

When the hydraulic pressure is generated by the gear pump **12**, it is supplied to the hydraulic motor **30** through a hydraulic channel **21** formed in the housing **20** of the drive part B or to the hydraulic motor **30** through a separate hydraulic hose, so that a motor shaft **31** of the hydraulic motor **30** generates the rotational force in forward/backward directions as illustrated in FIG. 2.

The hydraulic motor **30** is fixedly installed at a motor fixing plate **32** attached to a front end of one end of the housing **20** of the drive part B. A motor disk **34** to which a number of rotation pins **33** are attached in a radial shape is connected to the motor shaft **31** in a single unit. The motor disk **34** operates to rotate in the forward/backward directions as the hydraulic motor **30** operates. A motor casing **35** is installed at the outside of the hydraulic motor **30** and covers the hydraulic motor **30**, thereby protecting the hydraulic motor **30** from any external forces.

The hydraulic motor **30** generates high motive power by using low electric power. Further, as the hydraulic motor **30** uses an incompressible hydraulic fluid, it provides the effect of realizing much faster response characteristics, compared with a conventional electric actuation.

The decelerator **40** is positioned in the middle of the housing **20** of the drive part B. The decelerator **40** reduces a speed of the rotational force of the hydraulic motor **30** and transmits the rotational force to the driving shaft **44** opening/closing the valve and lock. As illustrated in FIG. 2, the decelerator **40** includes the worm shaft **43** rotatably installed in the middle of the housing **20** of the drive part B by bearings **41**. The worm **42** is formed about the worm shaft **43**. The worm **42** is engaged with the worm wheel **45** connected, in a single unit, to the driving shaft **44** being vertically installed. Therefore, the driving shaft **44** is variously rotated, in the forward/backward directions, at a reduced speed.

In the embodiment of the present invention, the decelerator **40** has the mutually connected constitution of the worm **42** and the worm wheel **45** but it may be realized in a different manner. Any element that is capable of reducing a rotational speed of the hydraulic motor **30** and transmitting the rotational force to the driving shaft **44** at the reduced speed is applicable within the scope of the present invention.

As illustrated in FIG. 3, the moving shaft **50** penetrates the worm shaft **43** so as to be connected to each other. Key grooves **51** are each formed at both sides of the moving shaft **50** and the worm shaft **43**. Each key **52** is installed in each key groove **51**, so that the moving shaft **50** is operatively rotated with the worm shaft **43** and is smoothly moved to slide to either side.

That is, the key **52** provides a connecting force between the worm shaft **43** and the moving shaft **50** and guides the moving shaft **50** to be smoothly moved by sliding horizontally.

When the automatic disk **60** is fitted into one side of the moving shaft **50** so as to be in a single unit, the each key **52** is inserted into the key groove formed in an inner diameter of the automatic disk **60**. Then, a number of rotation pins **63** formed in a radial shape are attached to the side of the automatic disk **60** facing the motor disk **34** in a single unit.

Thus, when the motor disk **34** is operated to rotate, the rotation pins **33** of the motor disk **34** and the rotation pins **63** of the automatic disk **60** meet each other in contact so as to be

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rotated. As a result, the rotational force transmitted to the worm **42** rotates the driving shaft **44** at a reduced speed.

As illustrated in FIG. 3, the manual disk **70** is fitted to be connected to the other side of the moving shaft **50**. The manual disk **70** functions as a medium to transmit the manual rotational force to the worm shaft **43** by selectively operating the manual handle **90**. The key **52** connected to the worm shaft **43** is simultaneously fitted into the key groove formed in an inner diameter of the manual disk **70**, so that the manual disk **70** is connected to the other side of the moving shaft **50** in a single unit.

In the manual mode switch **80** provided in the multi-turn hydraulic actuator according to the present invention, the release bar **85** positioned between the motor disk **34** and the automatic disk **60** selectively pivots as illustrated in FIGS. 4 and 5, to move the automatic disk **60** so as not to interfere with the motor disk **34** and to manually operate the worm shaft **43**.

That is, the manual mode switch **80** increases a distance between the rotation pins **33** of the motor disk **34** and the rotation pins **63** of the automatic disk **60** so that the rotation pins **33** do not interfere with the rotation pins **63**, thereby interrupting the motive power so that the rotational force of the hydraulic motor **30** is not transmitted to the worm shaft **43**. On the other hand, the manual mode switch **80** decreases a distance between the rotation pins **73** of the manual disk **70** and the rotation pins **93** of the handle disk **94** so that the rotation pins **73** and the rotation pins **93** meet each other, thereby changing the mode to the manual mode so that the actuator is operated by manipulating the manual handle **90**.

In the manual mode switch **80**, a manual switch housing **81** is installed to be adjacent to the motor disk **34** and the automatic disk **60** through the housing **20** of the drive part B. As illustrated in FIGS. 4 and 5, a pusher **83** is installed in the manual switch housing **81**. The pusher **83** is elastically installed by a pair of elastic members **82** positioned vertically as shown, to move up/down.

The pusher **83** includes an interference preventing groove **83a** formed in the middle of the pusher **83**. The interference preventing groove **83a** prevents the pusher **83** from contacting and interfering with a switch shaft **84** when the pusher **83** moves down. Pressing sides **83b** are formed by being extended from both lower sides of the interference preventing groove **83a**. A manual mode button **83c** exposed outward through the manual switch housing **81** is installed on an upper surface of the pusher **83**.

The switch shaft **84** penetrating the inside/outside of the manual switch housing **81** is installed in front of the pusher **83**, so as to be freely rotatable by a bearing **84a**. A pivot piece **84b** protrudes in an inner front end of the switch shaft **84**. The pivot piece **84b** pivots when it is pressed by the pusher **83**. The release bar **85** is attached to an outer front end of the switch shaft **84** in a single unit. A switch protrusion **85-1** and a roller **85-2** are respectively positioned at the opposite ends of the release bar **85**. The roller **85-2** pushes the automatic disk **60** to be forcibly moved. The release bar **85** is positioned between the motor disk **34** and the automatic disk **60**.

When the manual mode button **83c** is pressed, the pusher **83** is moved down so that the pressing sides **83b** press the pivot piece **84b**. The pivot piece **84b** pivots around the switch shaft **84**, irrespective of its original position, right or left, and simultaneously the release bar **85** is changed to be positioned in a horizontal direction as illustrated in FIG. 5. Then, the roller **85-2** pushes the automatic disk **60** to be forcibly moved, so that the rotation pins **63** of the automatic disk **60** are spaced apart from the rotation pins **33** of the motor disk **34** so as not to interfere with each other, thereby temporarily interrupting the transmission of the motive power.

An elastic element receiving groove **86** is formed in the manual switch housing **81**. The elastic element receiving groove **86** is connected to the switch shaft **84** at right angles. After a fixing ball **87** and an elastic member **82** are inserted into the elastic element receiving groove **86**, a set screw **88** is used to force the elastic member **82** against the fixing ball **87** so that the fixing ball **87** is tensioned by the elastic member **82**. Therefore, the fixing ball **87** is prevented from being loose and has elastic force. A ball receiving groove **89** is formed on an outer circumference of the switch shaft **84**. The fixing ball **87** is selectively fitted into the ball receiving groove **89**. Therefore, unless the rotational force is applied to the switch protrusion **85-1** of the release bar **85**, the release bar **85** is not changed in position due to the elastic force of the fixing ball **87** and is maintained in a fixed state.

Further, to change the mode of the actuator to the automatic mode, without any additional manipulation, a return protrusion **34-1** is attached to an outer circumference of the motor disk **34** as illustrated in FIG. 6. The return protrusion **34-1** protrudes outwardly. When the motor disk **34** is rotated by the operation of the hydraulic motor **30**, the return protrusion **34-1** applies an impact to the switch protrusion **85-1** of the release bar **85** positioned in the horizontal state, so that the release bar **85** is rotated to the original state and simultaneously the automatic disk **60** is released from being pushed.

As illustrated in FIG. 3, magnets **75** facing each other in the same pole direction are respectively buried in the sides of the manual disk **70** and the handle disk **94** facing each other, so that the manual disk **70** and the handle disk **94** have resistance. At the moment when the release bar **85** returns to the original state and the automatic disk **60** is released from being pushed, the moving shaft **50** is pushed towards the motor disk **34**. Therefore, the mode of the actuator is changed from the manual mode to the automatic mode by a simple method of operating the hydraulic motor **30**, without any special manipulation, thereby providing the conveniences in use.

As illustrated in FIG. 1, a handle casing **91** is attached to the other side of the housing **20** of the drive part B. The manual handle **90** is positioned to be rotatable in the handle casing **91**. When the mode of the actuator is changed to the manual mode, the manual handle **90** is to manually operate the worm shaft **43** so as to be rotated. In the manual handle **90**, the handle disk **94** is installed at a front end of the handle shaft **92**. A number of the rotation pins **93** are radially attached to the handle disk **94**. The rotation pins **93** and the rotation pins **73** of the manual disk **70** selectively meet each other, so that the manual disk **70** is operatively rotated to make it possible to manually operate the worm shaft **43** to be rotated.

To dedicatedly detect the rotational direction, the rotational speed and the extent of opening of the worm shaft **43**, the present invention provides a number of magnets **100-1** for detecting the extent of opening and a non-touch digital detector **100** for detecting the extent of opening, as illustrated in FIGS. 7 and 8.

As illustrated in FIG. 8, three magnets **100-1** for detecting the extent of opening are installed at an outer circumference of the worm shaft **43**, at the equal intervals of 120 degrees in the circumference direction, along a spiral direction. The non-touch digital detector **100** for detecting the extent of opening includes hole sensors **101** for perceiving signals of the magnets **100-1** for detecting the extent of opening and is installed to be received in a detector receiving unit **25** formed in the housing **20** of the drive part B as illustrated in FIG. 2.

The hole sensors **101** perceive signals of the magnets **100-1** for detecting the extent of opening three times during the worm shaft **43** rotates once, so that the measurement thereof is more minute and precise. A special low-power processor

102 is installed on the top of the non-touch digital detector **100** and calculates the rotational speed, the rotational direction and the extent of opening of the worm shaft **43**. After the low-power processor calculates these items, calculated results (signals) are transferred to the controller **13** installed in the operation part A or drive part B, through a signal line **103**. The controller **13** directly receives the signals detected by the non-touch digital detector **100** for detecting the extent of opening and controls the operation of the hydraulic pressure generator **10**.

Further, the controller **13** may output outwardly the signals detected by the non-touch digital detector **100** for detecting the extent of opening and simultaneously controls the operation of the hydraulic pressure generator **10** by control instructions from the outside.

Further, as illustrated in FIG. 7, a display **105** for displaying the extent of opening is assembled on the top of the non-touch digital detector **100** for detecting the extent of opening. The display **105** is capable of indicating the extent of opening by the power applied through the controller **13** of the operation part A at normal times and by the power applied through an additionally built-in battery **106** at the time when electricity fails. The display **105** is capable of easily setting/adjusting the accurate extent of opening by operating a button of the operation part A, without opening the housing.

The multi-turn hydraulic actuator according to the present invention can be installed in a narrow and small space. A user can install the operator part (the hydraulic pressure generator) in a space where it is easy to operate the multi-turn hydraulic actuator, without any limitation with respect to an installation place. Therefore, the multi-turn hydraulic actuator provides the effect of stably obtaining high outputs by using low power.

Furthermore, the multi-turn hydraulic actuator provides the effect of conveniently setting/adjusting the accurate extent of opening by operating only a button of the operation part A, without opening the housing when setting/adjusting the extent of opening, by using the non-touch digital detector for detecting the extent of opening.

While the present invention has been particularly shown and described with reference to exemplary embodiments thereof, it will be understood by those of ordinary skill in the art that various changes in form and details may be made therein without departing from the spirit and scope of the present invention as defined by the following claims.

What is claimed is:

1. A multi-turn hydraulic actuator comprising:

a hydraulic motor installed at a front end of one side of a housing of a drive part, to operate a motor disk so as to rotate by hydraulic pressure generated in a hydraulic pressure generator;

a decelerator installed in the middle of the housing of the drive part and including a worm, a worm shaft, a driving shaft and a worm wheel, wherein the worm shaft with the worm is installed to be rotatable, and the worm is engaged with the worm wheel connected to the driving shaft in a single unit, to reduce a speed of a rotational force of the worm and transmit the rotational force of the worm to the driving shaft at the reduced speed;

a moving shaft penetrating the worm shaft and connected to the worm shaft, to operatively rotate together and to selectively slide left or right;

an automatic disk connected to one side of the moving shaft and operatively rotated by a number of rotation pins selectively meeting rotation pins of the motor disk;

a manual disk connected to the other side of the moving shaft;

a manual mode switch including a release bar positioned between the motor disk and the automatic disk, wherein the release bar is pivoted when a manual mode button is operated and then the release bar moves the automatic disk so as not to interfere with the motor disk, to switch an automatic mode of the worm shaft to a manual mode; and

a manual handle installed at the other side of the housing of the drive part and including a handle shaft, rotation pins and a handle disk, wherein the rotation pins of the handle disk installed at a front end of the handle shaft selectively meet rotation pins of the manual disk, to manually rotate the worm shaft.

2. The multi-turn hydraulic actuator of claim 1, wherein the hydraulic pressure generator is installed, in a single unit or a separate unit, at the housing of the drive part directly connected to a valve of a conduit or a lock of a channel.

3. The multi-turn hydraulic actuator of claim 1, wherein the manual mode switch comprises:

a manual switch housing installed adjacently to the motor disk and the automatic disk through the housing of the drive part;

a pusher elastically installed to move upward/downward inside the manual switch housing by elastic members and including an interference preventing groove formed in the middle of the pusher, pressing sides formed to be extended to both lower sides from the interference preventing groove, and the manual mode button installed on an upper surface of the pusher;

a switch shaft installed to be rotatable through the inside and outside of the manual switch housing and including a pivot piece formed to protrude from an inside front end of the switch shaft, wherein the pivot piece pivots by being pressed by the pusher; and

the release bar attached to an outside front end of the switch shaft and including a switch protrusion and a roller respectively positioned at the opposite ends of the release bar, wherein the roller pushes the automatic disk to be forcibly moved.

4. The multi-turn hydraulic actuator of claim 3, wherein the manual switch housing includes an elastic element receiving groove connected to the switch shaft, wherein a fixing ball is elastically installed by an elastic member forced by a set screw so that the fixing ball is prevented from being loose, wherein the fixing ball is fitted to be inserted into a ball

receiving groove formed in an outer circumference of the switch shaft, so that the release bar is maintained in a fixed state.

5. The multi-turn hydraulic actuator of claim 1 or claim 4, wherein a return protrusion is formed to protrude outwardly at an outer circumference of the motor disk, wherein when the motor disk rotates as the hydraulic motor operates, the return protrusion applies an impact to the switch protrusion of the release bar positioned in a horizontal state so that the release bar returns to its original state.

6. The multi-turn hydraulic actuator of claim 5, wherein the manual disk and the handle disk include magnets facing each other in the same pole direction, so that the manual disk and the handle disk have resistance, wherein when the release bar returns to its original state and the moving shaft is released from being pushed, the magnets push the moving shaft towards the motor disk so that the actuator is automatically changed to the automatic mode.

7. The multi-turn hydraulic actuator of claim 1, further comprising:

a number of magnets for detecting the extent of opening, installed at an outer circumference of the worm shaft, at the equal intervals, along a spiral direction; and

a non-touch digital detector for detecting the extent of opening, installed to be received in a detector receiving unit formed in the housing of the drive part, wherein the non-touch digital detector includes hole sensors for perceiving signals of the magnets, to detect the rotational direction and the extent of opening of the worm shaft.

8. The multi-turn hydraulic actuator of claim 7, wherein the non-touch digital detector for detecting the extent of opening further includes a low-power processor and a display for indicating the extent of opening installed on the non-touch digital detector upper surface.

9. The multi-turn hydraulic actuator of claim 1, further comprising:

a controller provided in an operation part or drive part, wherein the controller controls the operation of the hydraulic pressure generator by directly receiving input signals detected by the non-touch digital detector for detecting the extent of opening or by outwardly outputting the signals detected by the non-touch digital detector for detecting the extent of opening and simultaneously receiving control instructions from the outside.

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