



SC, SD, SE, SG, SK, SL, SM, ST, SV, SY, TH, TJ, TM,
TN, TR, TT, TZ, UA, UG, US, UZ, VC, VN, ZA, ZM,
ZW.

EE, ES, FI, FR, GB, GR, HR, HU, IE, IS, IT, LT, LU,
LV, MC, MK, MT, NL, NO, PL, PT, RO, RS, SE, SI, SK,
SM, TR), OAPI (BF, BJ, CF, CG, CI, CM, GA, GN, GQ,
GW, KM, ML, MR, NE, SN, TD, TG).

(84) Designated States (*unless otherwise indicated, for every
kind of regional protection available*): ARIPO (BW, GH,
GM, KE, LR, LS, MW, MZ, NA, RW, SD, SL, SZ, TZ,
UG, ZM, ZW), Eurasian (AM, AZ, BY, KG, KZ, RU, TJ,
TM), European (AL, AT, BE, BG, CH, CY, CZ, DE, DK,

Published:

— *with international search report (Art. 21(3))*

APPARATUS FOR ALIGNING AND SECURING COMPONENTS OF A LIQUID COOLED PLASMA ARC TORCH

TECHNICAL FIELD

[01] Devices, systems, and methods consistent with the invention relate to cutting, and more specifically to devices, systems and methods for aligning and securing components of a liquid cooled plasma arc torch. More particularly, the invention relates to a coolant tube assembly and a cutting torch according to the preamble of claim 1 respectively 11.

BACKGROUND

[02] In many cutting operations, plasma arc torches are utilized. These torches operate at very high temperatures which can damage many components of the torches. As such, some torches use liquid cooling to transfer the heat away from some of the cutting torch components. The cooling liquid is passed through various fluid chambers, etc. However, the presence and need for these chambers and passages means that alignment of some of the components of the torch assembly can be difficult, especially when components are replaced. When installation alignment is poor the performance of the cooling can be adversely affected and thus the usable life of the torch and torch components can be greatly diminished. Some torches have added various stabilizing portions on some of the components that extend into the cooling fluid paths, however these stabilizing portions can interfere with fluid flow and thus compromise the cooling abilities of the torch assembly.

[03] Further limitations and disadvantages of conventional, traditional, and proposed approaches will become apparent to one of skill in the art, through comparison of such approaches with embodiments of the present invention as set forth in the remainder of the present application with reference to the drawings.

BRIEF SUMMARY OF THE INVENTION

[04] An exemplary embodiment of the present invention is an arc torch assembly or sub assembly having improved replacement and centering characteristics, where certain components of the torch head have particular characteristics which improve the operation, use and replaceability of the various components. Further embodiments are inferable from the ensuring description, drawings and claims.

BRIEF DESCRIPTION OF THE DRAWINGS

[05] The above and/or other aspects of the invention will be more apparent by describing in detail exemplary embodiments of the invention with reference to the accompanying drawings, in which:

[06] Fig. 1 illustrates an exemplary embodiment of a cutting torch coolant tube assembly of the present invention;

[07] Fig. 2 illustrates an another view of the cutting torch coolant tube of Figure 1;

[08] Figs. 2A and 2B illustrate a similar view of that shown in Figure 2, but of a different exemplary embodiment;

[09] Fig. 3 illustrates an exemplary embodiment of an thread pattern that can be used with various components of the present invention; and

[10] Fig. 4 illustrates an exemplary embodiment of a torch assembly utilizing the assembly of Figure 1.

DETAILED DESCRIPTION

[11] Exemplary embodiments of the invention will now be described below by reference to the attached Figures. The described exemplary embodiments are intended to assist the understanding of the invention, and are not intended to limit the scope of the invention in any way. Like reference numerals refer to like elements throughout.

[12] Figure 1 depicts a diagrammatical representation of an exemplary embodiment of a cutting torch cooling tube electrode assembly 100 of the present invention. As is generally understood, the assembly 100 is inserted into a torch body which is not shown here for clarity (see Figure 4). The assembly 100 comprises a coolant tube 101 which is inserted into a channel 109 of a coolant tube holder 105 and a channel 104 of an electrode 107. The distal end of the coolant tube holder 105 has an opening into which the electrode 107 is inserted. The proximate end of the holder 105 also has an opening into which the coolant tube 101 is inserted, as shown.

[13] The coolant tube 101 has a proximate end opening 103 which feeds into a channel 102 in the coolant tube. During operation, the cooling liquid is directed to the opening 103 and down through the channel 102 towards the distal end of the coolant tube 101. The tube 101 has a length such that its distal end creates a gap 111 between the end of the tube 101 and an inner wall of the channel 104 of the electrode 107. This gap 111 is important to the operation of the assembly 100 as the coolant flows down the channel 102 it passes through this gap 111 and enters the channel 104 of the electrode 107 and then the channel of the holder 105 to provide the desired cooling. Maintaining a consistent width of the gap 111 is important to proper coolant flow and in many known torch assemblies this is difficult to do, particularly when the electrode and/or coolant tube of prior torches is replaced. Because of the structure of known torches it is difficult to assemble the components to achieve the desired gap 111 dimension when replacing any of the components. This results in diminished cooling performance. Embodiments of the present provide for very consistent insertion of the tube 101 and the gap 111 dimension, as well as centering of the tube 101 in the channels 109 and 104, which will be described in more detail below.

[14] Once the coolant passes through the gap 111 it is directed through the channel 109 towards the proximate end of the holder 105 between the outer surface 110 of

the tube 101 and the inner surface 108 of the holder 105. In embodiments of the present invention, the holder 105 contains a plurality of exit ports 106 which allows the coolant to exit the channel 109 and transfer heat away from assembly 100. The ports 106 are positioned radially around a centerline of the holder 105 so that the coolant exits radially away from the holder 105 centerline as opposed to out of its proximate end. In exemplary embodiments, the holder 105 contains between 3 and 8 ports. The radial displacement of the ports is symmetrical to ensure even flow. The diameter of the ports is to be selected to ensure that the desired coolant flow is achieved during operation. In some exemplary embodiments all of the ports 106 have the same diameter. However, in other exemplary embodiments, the ports 106 can have different diameters. For example, half of the ports 106 can have a first diameter, while the other half of the ports 106 can have a second diameter which is less than the first diameter. Once the coolant exits the ports 106 it is recycled through a heat exchange and/or cooling system as is generally known and understood. Further, in some exemplary embodiments the ports have a circular opening, while in other exemplary embodiments, at least some of the ports 106 can have non-circular shapes like slots, etc. After cooling the electrode the coolant recirculates through the ports to a heat exchanger (not shown for clarity).

[15] Figure 2 shows a close up view of the proximate end of the coolant tube holder 105 and the coolant tube 101, which shows how the coolant tube 101 is stabilized and centered in the coolant tube holder 105. As shown, the coolant tube 101 has a stabilization portion 123 which extends radially around the tube 101. The stabilization portion 123 has an outer land surface 123A which engages with the inner surface 108 of the holder 105. When the tube 101 and the holder 105 are engaged with each other there is a friction fit engagement between the portion 123 and the surface 108. The friction fit engagement between the portion 123 and the surface 108 holds the tube 101 centered in the channel 109 and ensures that each time the cooling tube, and other components are replaced the

components are repositioned in a centered state with little difficulty. In exemplary embodiments, the portion 123 is configured such that the friction fit engagement with the holder 105 is continuous radially around the surface 108. Stated differently, the engagement between the portion 123 and surface 108 is such that not fluid (cooling fluid, etc.) can pass between the portion 123 and the surface 108. Thus, it is easier to replace the components, including the assembly 100 in a torch and providing more consistent accurate replacement.

[16] Another exemplary embodiment of the present invention, is shown in Figures 2A and 2B, where the coolant tube 101 has extension portions 140 which extend radially outward from the portion 123 as shown. These extension portions 140 extend out from portion 123 into grooves 108A in the coolant tube holder 105 and aide to ensure proper insertion into the coolant tube holder 105. In exemplary embodiments the extension portions 140 have a friction fit with the grooves 108A. This engagement aids in centering the coolant tube 101 as well as ensuring that the coolant tube 101 is oriented radially in the proper position. In exemplary embodiments, the extension portions 140 have a length which is less than the length L of the portion 123. Further, the extension portions have a surface 141 which engages with an adjacent surface 141A on the coolant tube holder 105. The engagement of these two surfaces acts to again ensure proper placement of the coolant tube 101 in the coolant tube holder 105 and ensure that it is not inserted too far into the holder 105. Although four portions 140 are shown in Figures 2A and 2B, other embodiments can use a different number of portions 140.

[17] In lieu of various aspects of the above described invention, the coolant tube 101 will always be inserted in a concentric state in its holder 105. Thus preventing improper insertion and decreased component life.

[18] Additionally, as shown the tube 101 has securing portion 119, which is closer to the proximal end of the tube than the stabilization portion 123, which is used in conjunction with a third portion 119A to hold an o-ring 130 in place. The o-ring 130 is used

to provide a seal for the assembly 100 and tube 101 when installed in a torch assembly. Each of the securing portion 119 and the third portion 119A extended radially around the tube 101. The securing portion 119 has a distal surface 122 which, when installed in the holder 105, engages with a the proximal end surface 120 of the holder 105. Because of this engagement, the insertion of the tube 101 into the holder 105 will always be made at the appropriate position to ensure that the gap 111 is the proper distance. In known torch assemblies the depth of insertion is difficult to repeat or perform consistently. Thus, the surfaces 122 and 120 ensure that the tube 101 is inserted to the proper distance easily and nearly eliminates error during replacement and assembly. Further, the combination of having the surface 122 engage with the surface 120 at the proximal end of holder 105 and the portion 123 engaging with the surface 108 provides a coolant tube assembly 100 with improved centricity and improved reliability during assembly and replacement of components over known torches. The combination of these engagements in close proximity to each other ensures that the tube 101 is inserted into the holder 105 at the proper depth for the gap 111 and centered within the channel 109. Further, this configuration allows the tube 101 to be configured without positional protrusions closer to the distal end of the tube 101. In some known torch assemblies the coolant tube has protrusions positioned closer to the distal end of the tube to aid in centering the tube. However, these protrusions extend into the coolant flow path and thus impede coolant flow and coolant performance. Some exemplary embodiments of the present invention can use positional protrusions, but because of the advantages of the above discussed configuration the protrusions can be smaller, and in many applications are not necessary.

[19] Also as shown in Figure 2, exemplary embodiments of the present invention include an undercut portion 133 positioned between portions 119 and 123. This undercut portion serves to ensure proper seating between surfaces 122 and 120 and thus the cool-

ant tube 101 in the coolant tube holder 105. This undercut portion 133 is to have a length along the coolant tube which is less than the length L of the portion 123.

[20] As described above, the stabilization portion 123 aids in stabilizing the tube 101 when inserted into the holder 105 in a press fit state. Thus, the length of the portion 123 needs to be sufficient to provide the desired stabilization and ensure centricity when inserted. To achieve this, in exemplary embodiments of the present invention, the outermost plateau surface 123A of the portion 123 has a length L that is in the range of 10 to 20% of the length of the tube 101 which is inserted into the holder 105 (the length of the tube from its distal end at the gap 111) . Having a plateau length in this range ensures sufficient alignment and stability while also allowing for accurate and repeatable positioning. In other exemplary embodiments the length of the plateau portion 123A is in the range of 4 to 25% of the length of the tube 101 within the holder 105. The plateau length L described above is the length of the flat surface on the portion 123 that makes contact with the inner surface of the holder 105 when the tube is inserted into the holder 105.

[21] As also shown in Figure 2, the portion 123 has an angled surface 123B which extends from the body of the tube 101 to the plateau surface 123A. The angled surface 123B aids in guiding the flow of the coolant fluid out of the ports 106. This aids in preventing the creation of stagnation zones in the fluid flow and increases the performance of the fluid flow. In some exemplary embodiments, the angle A between the body of the tube 101 and the surface 123B is in the range of 16 to 60 degrees. In other exemplary embodiments the angle is in the range of 40 to 60 degrees. Further, as shown in Figure 2, the center of the angle A is positioned such that it aligns with the centerline of the ports 106. If the angle A is a radiused angle A, as in some exemplary embodiments, then the center A corresponds to the center of a circle defined by the radius of the angle A, whereas if the angle A is a sharp angle then the center of the angle A is the inflection point. In some exemplary embodiments, the center of the angle A is aligned with the centerline of the ports

106. In other exemplary embodiments, the centerline of the angle A is positioned such that it is close to the centerline of the ports 106, but does not have to be aligned with the centerline. In such embodiments, the center of the angle A is positioned within 10% of the diameter of the ports 106 with respect to the centerline of the ports 106. For example, if the diameter of the ports 106 is 0.25", the center of the angle A is aligned within +/- 0.025" of the centerline of the ports. If the ports have varying diameters (as referenced previously) the average of the port diameters is to be used to determine the range of alignment as described above.

[22] As shown in Figure 1, the electrode 107 is shorter and threaded into the coolant tube assembly. Such a configuration allows the electrode 107 to be considerably smaller and much easier to be replaced. Because of this configuration, in exemplary embodiments of the present invention, the electrode 107 can have a length (from its most distal to most proximate ends) that is within the range of 4 to 20% of the coolant tube assembly 100, 5 to 20% of the length of the coolant tube 101, and within the range of 5 to 20% the length of the coolant tube holder 105. With these ratios, embodiments of the present invention provide excellent cutting performance and at the same time allow for ease of replacement and alignment of each of the respective components, as described herein. That is, when a component such as the electrode 107 need be replaced, the fit and construction of the assembly of the holder 105 and tube 101 (which can be replaced as a single unit) ensures proper replacement. Further, it is not necessary to remove the coolant tube holder and thus risk misaligning the coolant tube holder or the remainder of the assembly 100 when replacement of the electrode 105 is needed. Additionally, the coolant tube holder 105 and the coolant tube 101 can be kept as an assembly to be replaced as needed which ensures that the assembly

[23] The electrode 107 can be made of known materials used for electrodes, including but not limited to copper, silver, etc. Further, because of the reduced size of the

electrode 107 there is a significant reduction in cost by just replacing the electrode 107 of the present invention.

[24] Figure 3 depicts another aspect of the present invention, which aids in ensuring proper alignment and centricity during assembly and replacement of components of the assembly 100. Specifically, Figure 3 depicts a quick-coupling, multi-start thread configuration which is used on various components of the torch assembly 100, and can be used on other components of a torch. As described more fully below, the thread design employs multiple starts and a modified thread pitch to enhance alignment and installation, during assembly and replacement.

[25] As described previously, it is often necessary to remove and replace worn components of a cutting torch. Because of the need to replace components often it is desirable to speed up the process while at the same time ensuring that the replaced components are properly installed and aligned. Known torch assemblies use a standard single thread design, and some have used a bayonet thread design. However, these thread designs often require an appreciable number of turns to complete the installation, and increase the likelihood of an error during threading, such as cross-threading. For example, in most applications replacement of threaded components can require anywhere from 5 to 10 full turns of the item. By having such large number of turns for a component there is an increased likelihood of cross-threading the component, and/or result in the component not being completely tightened which can result in leaks and/or poor component life. Embodiments of the present invention address these issues by using a multi-thread design which utilizes existing required installation torque and thread stresses while maintaining the same applied force to mating parts as known thread systems.

[26] Figure 3 depicts an exemplary embodiment of an electrode 300 having a multi-thread design of the present invention. Specifically, the electrode 300 has a thread portion 301 having a plurality of separate and distinct thread paths 303A, 303B and 303C.

The embodiment shown has three distinct thread paths 303, but other embodiments of the present invention can use more than three thread paths. For example, other exemplary embodiments can use 4 distinct thread paths, and others can use as many as 5 different thread paths. By using multiple thread paths, embodiments of the present invention can provide easy and accurate replacement of components, greatly minimizing misalignment and/or cross-threading of components, while at the same time providing the required and desired applied connection force. Embodiments of the present invention, also deliver the desired mating force by using significantly less complete rotations of the component, thus making the replacement of a component quicker and more consistent. For example, embodiments of the present invention can provide the complete installation of a component with only 1 to 2 complete rotations of a component. In some exemplary embodiments, complete installation of a component can be achieved by 1.25 to 1.5 complete rotations of the component. For example, in certain applications electrodes of the present invention can be installed with only 1.25 to 1.5 complete rotations. By using such a low number of rotations to complete an installation, the chances of accurate and complete installation are greatly increased

[27] Thus, embodiments of the present invention can provide highly accurate installation by ensuring proper alignment, minimizing the chances of cross threading or misalignment and ensuring that the component (for example the electrode 107) is fully installed. By reducing the number of rotations required to install a component, embodiments of the present invention make it much easier on an installer to ensure that complete installation has been achieved. Because of the advantages of the present invention, the multi-thread configuration can be used on all components of a torch head assembly that utilize threads, and in particular those threads on components that are frequently replaced. For example, each of the threads 115, 117 and 127 shown in Figure 1 can have the multi-thread configuration as described above. Further, in addition to these components, em-

bodiments can also use this thread configuration on other torch assembly components, such as quick disconnect rings, inner and outer retaining caps, electrodes, coolant tubes, holders, etc. As shown in Figure 4, the torch attachment ring 401 connects the torch head to the torch base, the outer retaining cap 403 aids in retaining the torch shield cap and the inner retaining cap 405 aids in retaining the torch nozzle.

[28] Figure 4 depicts an exemplary embodiment of a torch assembly 400 that contains the assembly 100 from Figure 1. Because the other components of the torch assembly 400 are generally known, they are not discussed in detail herein. Of course, various embodiments of the present invention are not limited to the configuration of the torch assembly 400 as shown in Figure 4, or the assembly 100 as shown in Figures 1 and 2, and these embodiments are intended to be exemplary.

[29] While the claimed subject matter of the present application has been described with reference to certain embodiments, it will be understood by those skilled in the art that various changes may be made and equivalents may be substituted without departing from the scope of the claimed subject matter. In addition, many modifications may be made to adapt a particular situation or material to the teachings of the claimed subject matter without departing from its scope. Therefore, it is intended that the claimed subject matter not be limited to the particular embodiment disclosed, but that the claimed subject matter will include all embodiments falling within the scope of the appended claims.

REFERENCE NUMBERS

100	assembly	123A	surface
101	coolant tube	123B	surface
102	channel	127	thread
103	end opening	130	o-ring
104	channel	133	undercut portion
105	coolant tube holder	140	portion
106	port	141	surface
107	electrode	141A	surface
108	inner surface	300	electrode
108A	grooves	301	portion
109	channel	303	path
111	gap	303A	path
115	thread	303B	path
117	thread	303C	path
119	portion	400	torch assembly
119A	portion	401	ring
120	surface	405	cap
122	surface		
123	stabilizationportion		

CLAIMS

1. A coolant tube assembly (100) for a torch, comprising:

a coolant tube holder (105) having a distal end and a proximate end, each of said ends having an opening to create a channel (109) in said coolant tube holder (105), where said coolant tube holder channel has an inner surface; and

a coolant tube (101) inserted into said channel of said coolant tube holder (105), where said coolant tube (101) is inserted into said opening (103) at said proximate end of said coolant tube holder (105), where said coolant tube (101) has a distal end and a proximate end and each of said ends of said coolant tube (101) has an opening to create a channel in said coolant tube (101); characterized in that

said coolant tube (101) further comprises a stabilization portion (123) which extends radially around said coolant tube (101) and said stabilization portion (123) has an outer surface which engages with said inner surface of said coolant tube holder (105) and said engagement centers said coolant tube in said coolant tube holder (105), where said engagement is a sealed engagement such that no fluid can pass by said stabilization portion (123) when engage with said inner surface of said coolant tube holder (105);

said coolant tube (101) further comprises a securing portion which is closer to said proximate end of said coolant tube than said stabilization portion (123), said securing portion has a distal surface which engages with an end surface on said proximate end of said coolant tube holder (105);

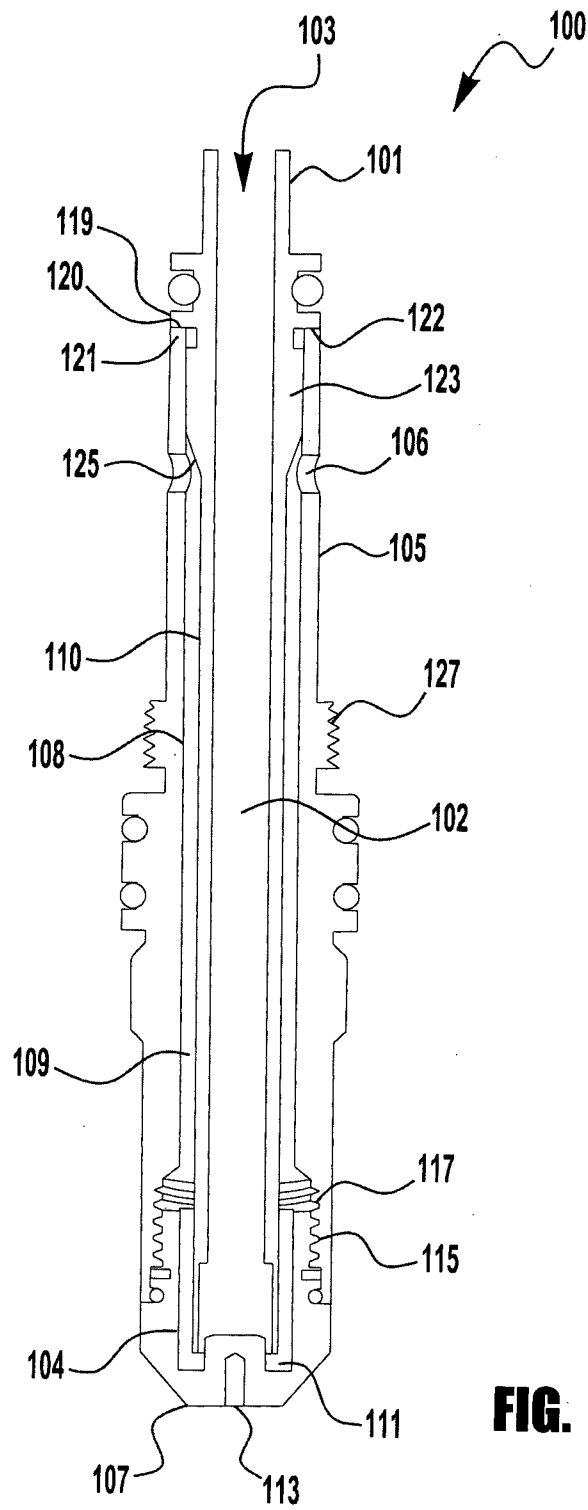
said coolant tube holder (105) further comprises a plurality of exits ports (106), where said exits ports (106) are positioned closer to the distal end of said coolant tube holder than said engagement between said stabilization portion (123) and said coolant tube holder (105) and said exit ports (106) are in communication with said coolant tube holder channel (109); and

said coolant tube (101) comprises an undercut portion between said stabilization portion (123) and said securing portion creating a gap (111) between said inner surface of said coolant tube holder (105) and said coolant tube (101).

2. The coolant tube assembly of claim 1, further comprising an electrode coupled to a distal end of said coolant tube holder.
3. The coolant tube assembly of claim 2, wherein said electrode has a length which is the range of 4 to 20% of the length of the assembled said coolant tube and said coolant tube holder.
4. The coolant tube assembly of one of the claims 1 to 3, wherein some of said exit ports have a first diameter and other of said exit ports have a second diameter.
5. The coolant tube assembly of one of the claims 1 to 4, wherein said outer surface of said stabilization portion which engages with said inner surface of said coolant tube holder has a length which is in the range of 4 to 25% or 10 to 20% of the length of the coolant tube which is inserted into the coolant tube holder.
6. The coolant tube assembly of one of the claims 1 to 5, wherein a distal end of said stabilization portion comprises an angled surface from said coolant tube to said outer surface of said stabilization portion, and said angled surface having an angle between its surface and the coolant tube in the range of 16 to 60 degrees, or in the range of 40 to 60 degrees.

7. The coolant tube assembly of claim 6, wherein a center of the angle between said coolant tube and said outer surface of said stabilization portion is positioned, in a length-wise direction along said coolant tube holder, within 10% of the diameter of the exit ports with respect to a centerline of said exit ports.
8. The coolant tube assembly of one of the claims 1 to 7,
wherein a distal end of said stabilization portion comprises an angled surface from said coolant tube to said outer surface of said stabilization portion.
9. The coolant tube assembly of claim 8, wherein said angled surface having an angle between its surface and the coolant tube in the range of 16 to 60 degrees.
10. The coolant tube assembly of claim 8 or 9, wherein said angle is in the range of 40 to 60 degrees.
11. A cutting torch characterized by the coolant tube assembly of one of the claims 1 to 10.

1/5



2/5

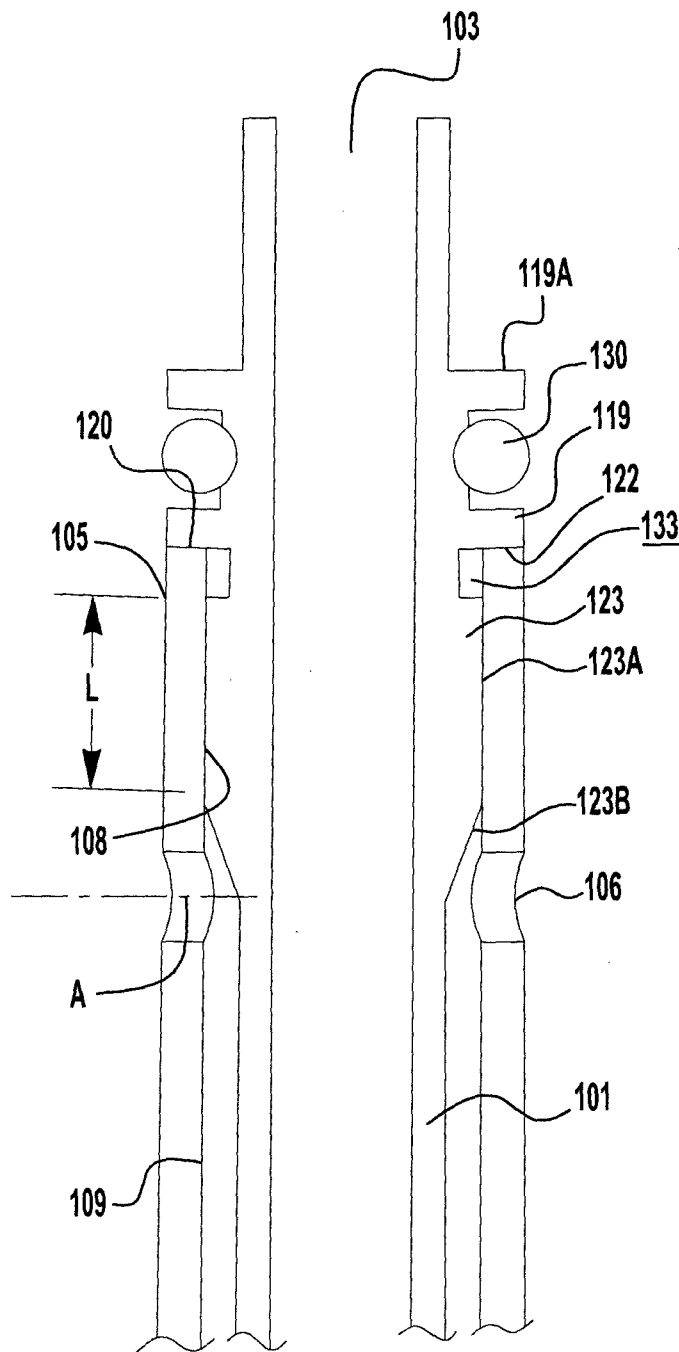


FIG. 2

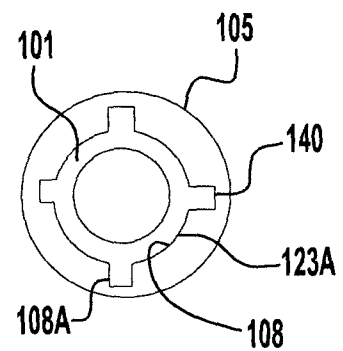


FIG. 2A

3/5

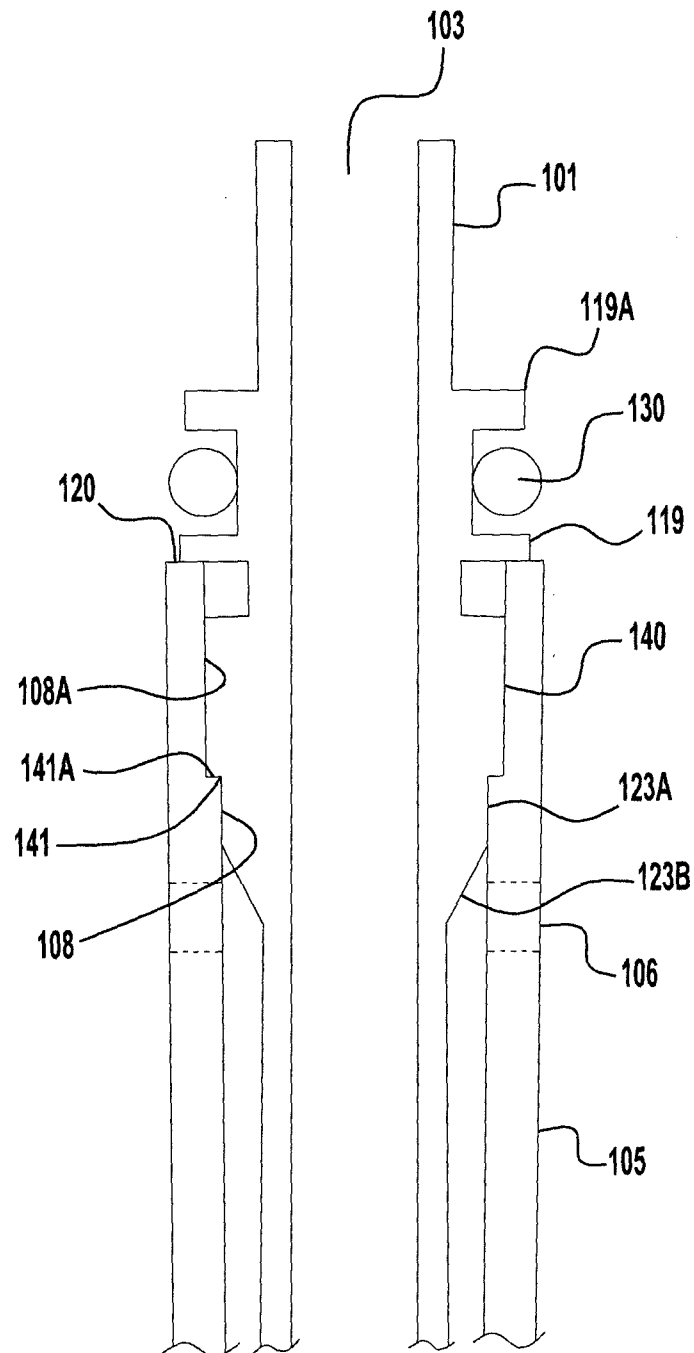


FIG. 2B

4/5

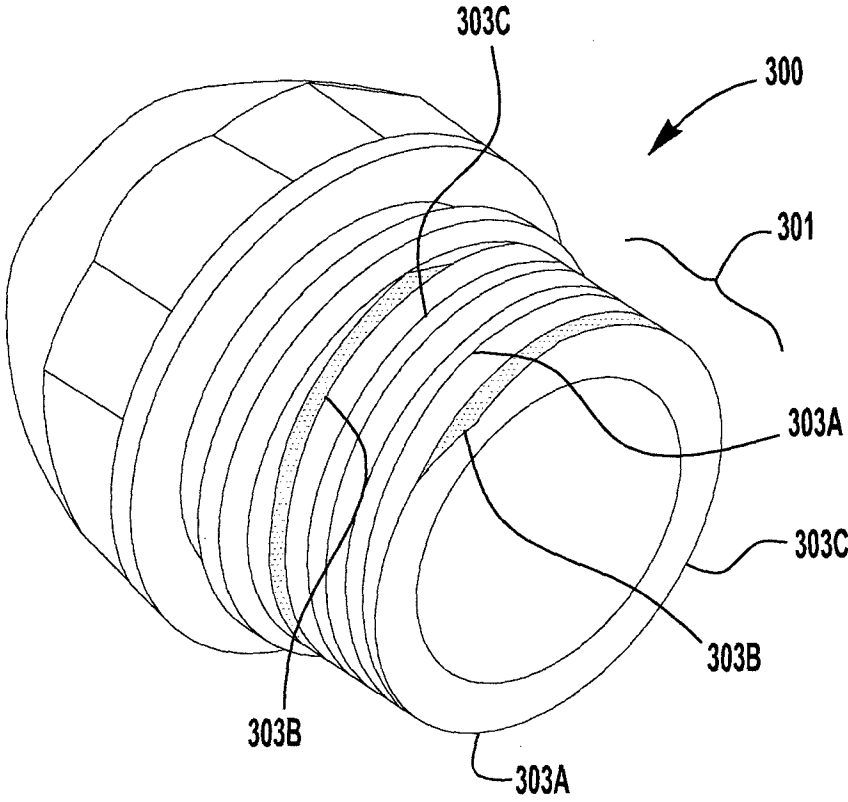


FIG. 3

5/5

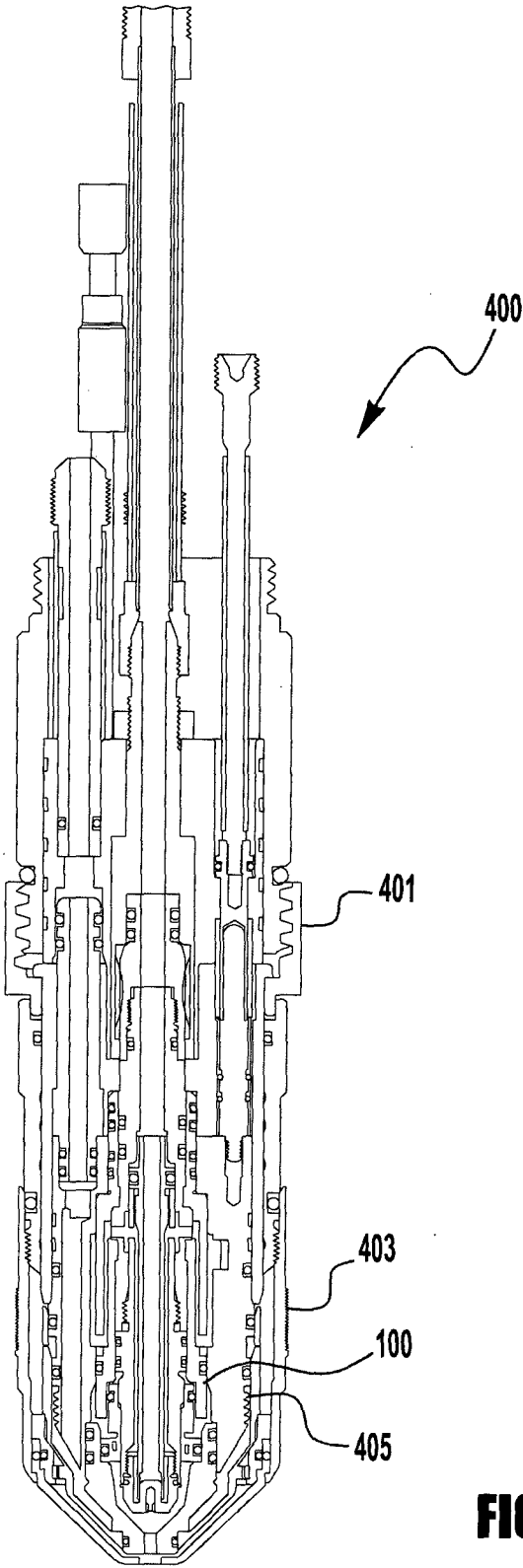


FIG. 4

INTERNATIONAL SEARCH REPORT

International application No

PCT/IB2014/001354

A. CLASSIFICATION OF SUBJECT MATTER
 INV. H05H1/28 H05H1/34
 ADD.

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)
 H05H

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

EPO-Internal, WPI Data

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 5 278 388 A (HUANG HUANG-NAN [TW]) 11 January 1994 (1994-01-11) abstract; figures 1,2 column 1, line 63 - column 3, line 21 -----	1-11
X	US 5 414 237 A (CARKHUFF DONALD W [US]) 9 May 1995 (1995-05-09) abstract; figure 1 column 5, lines 33-41 -----	1-3,5-11
X	US 4 967 055 A (RANEY TY A [US] ET AL) 30 October 1990 (1990-10-30) abstract; figures 1,3,4,5 column 5, line 67 - column 6, line 62 -----	1-3,5-11
X	US 2004/200810 A1 (BRANDT AARON D [US] ET AL) 14 October 2004 (2004-10-14) abstract; figures 3,7B,11 ----- -/-	1,6-11



Further documents are listed in the continuation of Box C.



See patent family annex.

* Special categories of cited documents :

"A" document defining the general state of the art which is not considered to be of particular relevance

"E" earlier application or patent but published on or after the international filing date

"L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)

"O" document referring to an oral disclosure, use, exhibition or other means

"P" document published prior to the international filing date but later than the priority date claimed

"T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention

"X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone

"Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art

"&" document member of the same patent family

Date of the actual completion of the international search

12 November 2014

Date of mailing of the international search report

19/11/2014

Name and mailing address of the ISA/

European Patent Office, P.B. 5818 Patentlaan 2
 NL - 2280 HV Rijswijk
 Tel. (+31-70) 340-2040,
 Fax: (+31-70) 340-3016

Authorized officer

Crescenti, Massimo

INTERNATIONAL SEARCH REPORT

International application No

PCT/IB2014/001354

C(Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	US 2010/155373 A1 (YAMAGUCHI YOSHIHIRO [JP] ET AL) 24 June 2010 (2010-06-24) abstract; figures 2,3 -----	6-9

INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No

PCT/IB2014/001354

Patent document cited in search report	Publication date	Patent family member(s)	Publication date
US 5278388	A	11-01-1994	EP 0629106 A1 14-12-1994 US 5278388 A 11-01-1994
US 5414237	A	09-05-1995	CA 2133498 A1 15-04-1995 JP 2552257 B2 06-11-1996 JP H07164152 A 27-06-1995 US 5414237 A 09-05-1995
US 4967055	A	30-10-1990	NONE
US 2004200810	A1	14-10-2004	AT 492144 T 15-01-2011 AU 2004229670 A1 28-10-2004 BR PI0409268 A 28-03-2006 CA 2521009 A1 28-10-2004 CN 1771766 A 10-05-2006 CN 101579778 A 18-11-2009 DE 202004021644 U1 10-09-2009 EP 1621052 A1 01-02-2006 EP 2265098 A2 22-12-2010 EP 2271190 A2 05-01-2011 ES 2353307 T3 01-03-2011 IN 219017 A1 13-06-2008 JP 5105469 B2 26-12-2012 JP 2006523006 A 05-10-2006 KR 20060023955 A 15-03-2006 KR 20090032125 A 31-03-2009 MX PA05010756 A 25-05-2006 US 2004200810 A1 14-10-2004 US 2005092718 A1 05-05-2005 US 2006151447 A1 13-07-2006 US 2007045245 A1 01-03-2007 US 2009308849 A1 17-12-2009 WO 2004093502 A1 28-10-2004
US 2010155373	A1	24-06-2010	CN 101801583 A 11-08-2010 JP 5205377 B2 05-06-2013 JP 5567071 B2 06-08-2014 JP 2012192456 A 11-10-2012 KR 20090131291 A 28-12-2009 US 2010155373 A1 24-06-2010 WO 2009008271 A1 15-01-2009