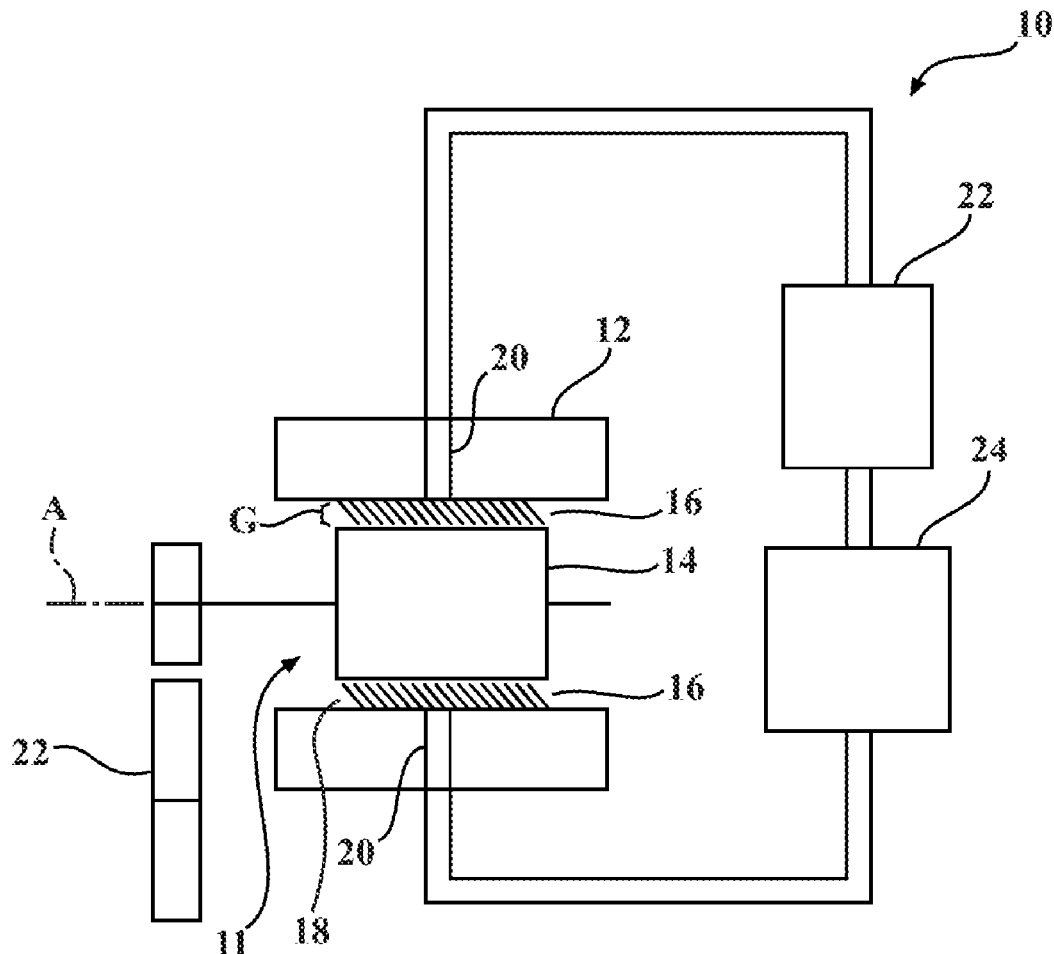




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Remboski et al.(10) **Pub. No.: US 2023/0001738 A1**(43) **Pub. Date: Jan. 5, 2023**(54) **LUBRICANT SUPPORTED ELECTRIC
MOTOR WITH WHEEL SUPPORT***F16H 57/04* (2006.01)*H02K 7/00* (2006.01)(71) Applicant: **Neapco Intellectual Property
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7/006 (2013.01)(72) Inventors: **Donald Remboski**, Ann Arbor, MI
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Oostkamp (BE)(57) **ABSTRACT**(21) Appl. No.: **17/940,018**(22) Filed: **Sep. 8, 2022****Related U.S. Application Data**(63) Continuation of application No. 16/668,390, filed on
Oct. 30, 2019, now Pat. No. 11,472,226.(60) Provisional application No. 62/752,442, filed on Oct.
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A lubricant supported electric motor includes a stator extending along an axis, and a rotor rotatably disposed around the stator in radially surrounding and spaced relationship to define at least one support chamber. A lubricant is disposed in the support chamber for supporting the rotor around the stator. A wheel rim is fixedly attached to the rotor and is disposed in surrounding relationship with the rotor and the stator. Thus, in a first aspect, rotation of the rotor is directly transferred to the wheel rim such that the wheel rim rotates in accordance with the rotation of the rotor. In accordance with another aspect, the rotor is rotatably disposed within the stator, and a planetary gear reduction mechanism is operably interconnected to the rotor, the stator, and the wheel rim and configured to rotate the wheel rim in response to rotation of the rotor within the stator.





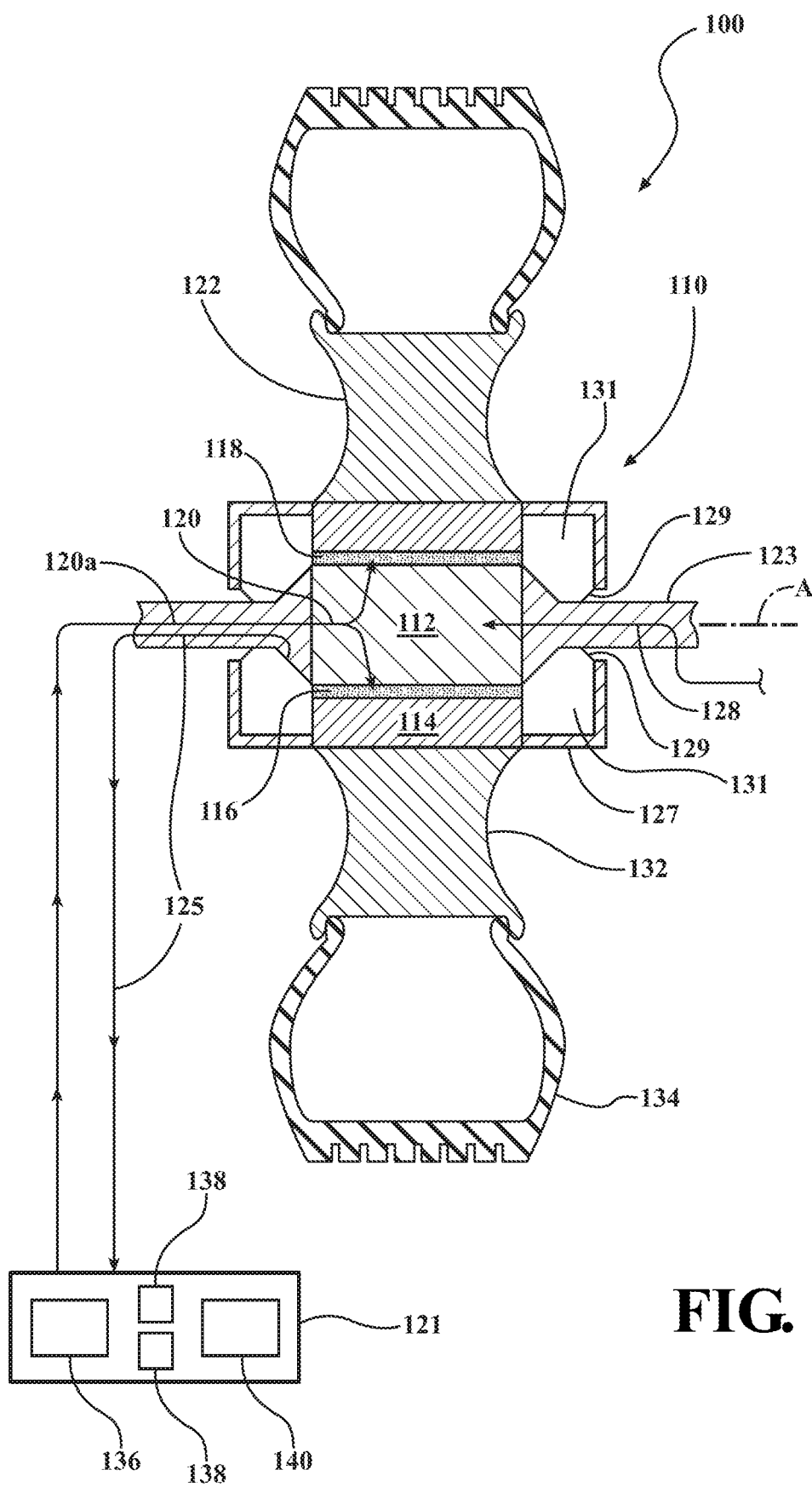


FIG. 2

LUBRICANT SUPPORTED ELECTRIC MOTOR WITH WHEEL SUPPORT

CROSS REFERENCE TO RELATED APPLICATION

[0001] This application is a continuation of U.S. application Ser. No. 16/668,390 filed on Oct. 30, 2019, which claims the priority and benefit of U.S. Provisional Application Ser. No. 62/752,442 filed on Oct. 30, 2018, the entire disclosures of which are hereby incorporated by reference.

FIELD OF THE DISCLOSURE

[0002] The present disclosure relates generally to a lubricant supported electric motor. More specifically, the present disclosure relates to a lubricant supported electric motor with an integrated wheel support.

BACKGROUND OF THE INVENTION

[0003] This section provides a general summary of background information and the comments and examples provided in this section are not necessarily prior art to the present disclosure.

[0004] Various drivelines in automotive, truck, and certain off-highway applications take power from a central prime mover and distribute the power to the wheels using mechanical devices such as transmissions, transaxles, propeller shafts, and live axles. These configurations work well when the prime mover can be bulky or heavy, such as, for example, various internal combustion engines (“ICE”). However, more attention is being directed towards alternative arrangements of prime movers that provide improved environmental performance, eliminate mechanical driveline components, and result in a lighter-weight vehicle with more space for passengers and payload.

[0005] “On wheel”, “in-wheel” or “near-wheel” motor configurations are one alternative arrangement for the traditional ICE prime mover that distributes the prime mover function to each or some of the plurality of wheels via one or more motors disposed on, within, or proximate to the plurality of wheels. For example, in one instance, a traction motor, using a central shaft through a rotor and rolling element bearings to support the rotor, can be utilized as the “on wheel”, “in wheel” or “near wheel” motor configuration. In another instance, a lubricant supported electric motor, such as described in U.S. application Ser. No. 16/144,002, can be utilized as the “on wheel”, “in wheel” or “near wheel” motor configuration. While each of these motor configurations result in a smaller size and lighter weight arrangement as compared to the prime movers based on the internal combustion engine, they each have certain drawbacks and disadvantages.

[0006] For example, the utilization of traction motors as the “on wheel”, “in wheel” or “near wheel” configuration still results in motors that are too heavy and not robust enough to shock loading to be useful for wheel-end applications. In other words, present traction motors are large, heavy structures supported by rolling element bearings, which are too heavy and large to be practical for wheel end applications.

SUMMARY OF THE INVENTION

[0007] In accordance with one aspect, the subject invention is directed to a lubricant supported electric motor

including a stator extending along an axis, and a rotor extending along the axis and rotatably disposed around the stator in radially surrounding and spaced relationship to define at least one support chamber extending between the stator and the rotor. A lubricant is disposed in the at least one support chamber for supporting the rotor around the stator. A wheel rim is fixedly attached to the rotor and is disposed in surrounding relationship with the rotor and the stator. Thus, in this aspect, rotation of the rotor is directly transferred to the wheel rim such that the wheel rim rotates in accordance with the rotation of the rotor. In other words, the rotor of the lubricant supported electric motor directly supports a driven wheel.

[0008] In accordance with another aspect, the subject invention is directed to a lubricant supported electric motor includes a stator extending along an axis, and a rotor extending along the axis and rotatably disposed radially within the stator in spaced relationship to define at least one support chamber extending between the stator and rotor. A lubricant is disposed in the at least one support chamber for supporting the rotor within the stator. A wheel rim is disposed in radially surrounding relationship with the stator and the rotor, and a planetary gear reduction mechanism is operably interconnected to the rotor, the stator, and the wheel rim and configured to rotate the wheel rim in response to rotation of the rotor within the stator.

[0009] The lubricant supported electric motor in either of these aspects is light and small, and thus contributes to the overall design strategy for eliminating weight and size from automobiles and land vehicles. Other advantages will be appreciated in view of the following more detailed description of the subject invention.

BRIEF DESCRIPTION OF THE DRAWINGS

[0010] The drawings described herein are for illustrative purposes only of selected embodiments and not all possible implementations, and are not intended to limit the scope of the present disclosure.

[0011] FIG. 1 is a schematic view of a lubricant supported electric motor according to the subject disclosure;

[0012] FIG. 2 is a cross-sectional view of a first aspect of the lubricant supported electric motor illustrating a directly supported wheel; and

[0013] FIG. 3 is a cross-sectional view of another aspect of the lubricant supported electric motor illustrating a wheel end motor with a reduction mechanism and a wheel supported by a ring gear of the reduction mechanism.

DETAILED DESCRIPTION OF THE ENABLING EMBODIMENTS

[0014] Example embodiments of a lubricant supported electric motor with integrated wheel support will now be more fully described. Each of these example embodiments are provided so that this disclosure is thorough and fully conveys the scope of the inventive concepts, features and advantages to those skilled in the art. To this end, numerous specific details are set forth such as examples of specific components, devices and mechanisms associated with the lubricant supported electric motor to provide a thorough understanding of each of the embodiments associated with the present disclosure. However, as will be apparent to those skilled in the art, not all specific details described herein need to be employed, the example embodiments may be

embodied in many different forms, and thus should not be construed or interpreted to limit the scope of the disclosure.

[0015] FIG. 1 illustrates a lubricant supported electric motor 10 in accordance with an aspect of the disclosure. As best illustrated in FIG. 1, the lubricant supported electric motor 10 includes a stator 12 and a rotor 14 extending along an axis A and movably disposed within the stator 12 to define a support chamber or gap 16 therebetween. A lubricant 18 is disposed in the gap 16 for supporting the rotor 14 within the stator 12, and providing continuous contact between these components. The lubricant 18 may therefore act as a buffer (e.g., suspension) between the rotor 14 and stator 12 minimizing or preventing contact therebetween. In other words, the lubricant 18 prevents direct contact between the stator 12 and rotor 14 and provides an electric lubricant supported motor 10 which is robust to shock and vibration loading due to the presence of the lubricant 18. Additionally, and alternatively, a substantially incompressible lubricant 18 may be used in order to minimize the gap between the stator 12 and rotor 14.

[0016] As further illustrated FIG. 1, the stator 12 defines a passageway 20 disposed in fluid communication with the gap 16 for introducing the lubricant 18. However, the passageway 20 could be provided on any other components of the lubricant supported electric motor 10 without departing from the subject disclosure. According to an aspect, the lubricant 18 may be cycled or pumped through the passageway 20 and into the gap 16 in various ways. For example, a high pressure source (e.g., a pump) 22 of the lubricant 18 may be fluidly coupled to a low pressure source (e.g., a sump) 24 of the lubricant 18, where the lubricant may move from the high pressure source to the lower pressure source, through the passageway 20 and into the gap 16. Rotation of the rotor 14 relative to the stator 12 may operate as a self-pump to drive lubricant 18 through the passageway 20 and into the gap 16.

[0017] As further illustrated in FIG. 1, the rotor 14 is interconnected to a drive assembly 22 for coupling the lubricant supported electric motor 10 to one of the plurality of wheels of a vehicle. For example, in one instance, the drive assembly 22 may include a planetary gear system. Alternatively, the drive assembly 22 may include one or more parallel axis gears. The stator 12 and rotor 14 are configured to exert an electromagnetic force therebetween to convert electrical energy into mechanical energy, moving the rotor 14 and ultimately driving the wheel coupled to the lubricant supported electric motor 10 via the drive assembly 22. The drive assemblies 22 may provide one or more reduction ratios between the lubricant supported electric motor 10 and the wheel in response to movement of the rotor 14.

[0018] The above described aspect of the disclosure illustrated in FIG. 1 is directed to the illustrated embodiment in which the stator 12 surrounds the rotor 14. However, the general operation of the lubricant 18 and the relationship between the stator 12 and the rotor 14 may be used in alternative arrangements of the rotor 14 and stator 12, such as the rotor 14 being disposed radially outward from the stator 12, as further described below. In such instances, the wheel structure may be supported by the rotor 14, rather than the rotor 14 being connected to a drive assembly that drives the wheel.

[0019] With reference to FIG. 2, a wheel end electric motor system 100 of another aspect of the disclosure is

provided. The system 100 may include a wheel directly supported on a lubricant supported electric motor 110. This configuration may be desirable for single-track vehicles, such as scooters, where strong overturning moments on the wheel are reduced or non-existent.

[0020] The system 110 may include a stator 112 and a rotor 114 extending along an axis A movably disposed around the stator 112 to define a gap or support chamber 116 disposed therebetween. A lubricant 118 is disposed in the gap 116 for supporting the rotor 114 around the stator 112, and providing continuous contact between these components. The lubricant 118 may therefore act as a buffer (e.g., suspension) between the rotor 114 and stator 112 minimizing or preventing contact therebetween. In other words, the lubricant 118 prevents direct contact between the stator 112 and the rotor 114 and provides an electric lubricant supported motor 110 which is robust to shock and vibration loading due to the presence of the lubricant 118. Additionally, and alternatively, a substantially incompressible lubricant 118 may be used in order to minimize the gap between the stator 112 and the rotor 114.

[0021] As further illustrated FIG. 2, the stator 112 defines a passageway 120 disposed in fluid communication with the gap 116 for introducing the lubricant 118. However, the passageway 120 could be provided on any other components of the lubricant supported electric motor 110 without departing from the subject disclosure. According to an aspect, the lubricant 118 may be cycled or pumped through the passageway 120 and into the gap 116 in various ways. For example, a high pressure source 121 (e.g., a pump) of the lubricant 118 may be fluidly coupled to a low pressure source (e.g., a sump, not shown) of the lubricant 118, where the lubricant may move from the high pressure source to the lower pressure source, through the passageway 120 and into the gap 116. Rotation of the rotor 114 relative to the stator 112 may operate as a self-pump to drive lubricant 118 through the passageway 120 and into the gap 116.

[0022] The stator 112 may be attached to or integrated with an axle 123 that extends coaxially with the axis A. The axle 123 may provide a channel 120a through which the lubricant 118 may pass between the pump 121 and the passageway 120 of the stator 112 that provides the lubricant 118 to the gap 116. The axle 123 may further define an outlet 125 in fluid communication with the gap 116. During operation of the motor 110, the lubricant 118 may be drained from the gap 116 and directed back toward the pump 121, with fresh lubricant 118 replacing the used lubricant 118. Accordingly, the lubricant 118 may be cycled through the motor 110.

[0023] The lubricant supported electric motor 110 may further include a housing or casing 127 that surrounds the stator 112. The casing 127 may be attached to the rotor 114, and may therefore rotate along with the rotor 114 during operation of the motor 110. The casing 127 may further include a seal portion 129 that interfaces with the axle 123 and/or stator 112. The casing 127 therefore defines an internal cavity 131, which is disposed in fluid communication with and may include the gap 116. The gap 116 generally refers to the area radially between the rotor 114 and stator 112, but the gap 116 is in fluid communication with the remainder of the cavity 131, and lubricant in the gap 116 may flow freely into the remainder of the cavity 131. The above described outlet 125 may intersect the cavity 131 at a location axially outward from the gap 116. As the rotor

114 rotates and the casing **127** rotates, the seal portion **129** will generally bear against axle **123** and/or stator **112**, while retaining the lubricant **118** within the cavity **131**, such that the lubricant **118** will be limited to exiting the cavity via the outlet **125**.

[0024] The axle **123** is preferably connected to the vehicle suspension or chassis (not shown), and does not rotate. The axle **123** may include wiring channels or passageways **128** for receiving and routing wiring or the like that may transfer current to the stator **112**. The stator **112** includes windings or the like that receive a current for creating the electrical field that drives the rotor **114**. The current supplied to the stator **112** may be a phase current.

[0025] In the system **100** of FIG. 2, the rotor **114** is rotationally fixed to a wheel **122**. The wheel **122** may be in the form of a wheel rim **132** with an attached tire **134**, or the wheel **122** may include an outer surface designed to directly bear against a ground surface. The rotor **114** may be directly attached to the wheel **122**, or the rotor **114** may be fixedly attached to the wheel **122** via intermediate structure. In the system of FIG. 2, the rotor **114** and wheel **122** are directly attached, permitting direct-drive of the wheel **122** from the motor **110**.

[0026] Optionally, the system **110** may include a suspension element disposed radially between the rotor **114** and the wheel **122** to provide a damping feature. This suspension material may be in the form of a compliant wheel structure, and allow the wheel **122** to shift radially relative to the rotor **114**, which acts as a hub. This additional damping material may allow for the system **100** to be used in various vehicles, such as electric bikes or golf carts, such that other suspension components typically attached to the axle **123** may be eliminated or reduced.

[0027] As described above, the lubricant **118** may delivered to the gap **116** (and also the cavity **131**) through the axle **123** and stator **112**. The stator **112** may include lubricant channels **120** therein that communicate with corresponding lubricant channels **120a** in the axle **123**. The lubricant **118** cycles through the system **100**, where it drains from the gap **116** and cavity **131** through the outlet **125** in the axle **123** and returns to the pump **121**. The pump **121** may include a known mechanism for pumping fluid, and the pump **121** may further include additional components for treating the lubricant **118**, such as a thermal control mechanism **136** that may cool or heat the lubricant to a desired temperature to control viscosity. The thermal control mechanism **136** may include sensors **138** and a controller **140** for managing the lubricant temperature. The pump **121** may further include a filter mechanism that filters the lubricant **118** to remove impurities and the like. The pump **121** may include sensors associated with the filter mechanism to measure the status of the fluid and/or filter.

[0028] The above described system **100**, providing wheel support on the electric motor **110**, reduces the overall size of the wheel end system, such that the motor **110** does not need to be placed beside the wheel hub to drive the wheel.

[0029] With reference to FIG. 3, in another aspect, a system **200** includes a lubricant supported electric motor **210** having a stator **212** and a rotor **214**. In this aspect, the rotor **214** is disposed within the stator **212**, similar to the arrangement of the lubricant supported electric motor **10** shown in FIG. 1. However, unlike the lubricant supported motor **10** of

FIG. 1, gear reduction and wheel support is provided by structure surrounding the rotor **214**, which is further described below.

[0030] The lubricant supported electric motor **210** includes a gap or support chamber **216** disposed between radially between the rotor **214** and stator **212**, with the gap **116** configured to receive lubricant **218** to support the rotor **214** within the stator **212**. The lubricant **218** within the gap **216** may also drain out of the gap **216** and be cycled through the system **210** to a pump or the like (not shown in FIG. 3), similar to the pump **121** of the system **100**.

[0031] The lubricant supported electric motor **210** further includes a gear reduction mechanism **222**. The gear reduction mechanism **222** may be in the form of a planetary gear reduction mechanism, in which a number of circumferentially fixed planet gears **P** are disposed around a rotatable sun gear **S**, which rotates the planet gears **P** about their individual axes, causing a ring gear **R** that surrounds the planet gears **P** to rotate at a different rotational velocity than the sun gear **S**.

[0032] The planet gears **P** are attached to a planet carrier **223**, which may be fixedly attached to a vehicle chassis. The planet carrier **223** may be considered a replacement for an axle type structure, or the chassis may include an axle portion that is fixed to the planet carrier **223**. The planet carrier **223** remains in a generally fixed position relative to the axle/chassis of the vehicle. The planet carrier **223** may include a pair of outer body portions **223a** that support a plurality of circumferentially arranged pins **223b**. The pins **223b** remain in a generally fixed position relative to the planet carrier **223**, and the pins **223b** support the planet gears **P** for rotation. Thus, each of the individual planet gears **P** may rotate about an axis defined by the pin **223b** on which they are supported.

[0033] The planet carrier may further include an inner body portion **223c**. The inner body portion **223c** may have the same general axial location as the rotor **214** and stator **212**, and the inner body portion **223c** may have an annular shape that supports the pins **223b**, similar to the outer body portion **223a**. Thus, the pins **223b** extend axially between the outer body portion **223a** and the inner body portion **223c**. The inner body portion **223c**, along with the outer body portion **223a** and the pins **223b**, remains stationary during operation of the motor **210**.

[0034] The stator **212** is mounted to or otherwise fixedly attached to the inner body portion **223c**. The stator **212** may be integrally formed with the inner body portion **223c**, or it may be separate component. The stator **212**, including the windings and the like, may be disposed radially inward from the inner body portion **223c**, such that the stator **212** is radially between the inner body portion **223c** and the rotor **214**. In an alternative approach, the stator **212** may be mounted or attached to the inner body portion **223c** such that the inner body portion **223c** is disposed radially between the stator **212** and the rotor **214**. In this approach, the distance between the rotor **214** and the stator **212** is small enough that the current applied to the stator **212** will still effectively rotate the rotor **214**.

[0035] The stator **212** and the planet carrier **223** thereby combine to define a fixed structure, with the rotor **214** disposed within this fixed assembled structure. The lubricant **218** may be delivered into the lubricant supported electric motor **210** and drained from the motor **210** via fluid channels

224 disposed in both the planet carrier **223** and the stator **212**. The fluid channels **224** may extend through one or more of the pins **223b**.

[0036] The reduction mechanism **222** includes the ring gear **R** previously described above. The ring gear **R** has a generally annular shape and circumferentially and radially surrounds the planet gears **P**. The ring gear **R** therefore has inner teeth that engage outer teeth of the planet gears **P**. The ring gear **R** may be in the form of axially outer portions that are disposed on opposite axial sides of the rotor **214** and stator **212**. The ring gear **R** rotates in response to rotation of the planet gears **P**.

[0037] The ring gear **R** is fixedly attached to a wheel rim **225** that surrounds the inner body portion **223c**, stator **212**, and rotor **214**. The wheel rim **225** will therefore rotate around the stationary stator **212** and inner body portion **223c** of the planet carrier **223** in accordance with the rotation of the ring gear **R**. The ring gear **R** and wheel rim **225** will also rotate around the rotor **214**, which is disposed at the radial center of the motor **210**. The wheel rim **225** may support a tire attached thereto (not shown in FIG. 3), or the wheel rim **225** may include wheel structure configured to engage the ground in lieu of a separate tire.

[0038] As described above, the lubricant **218** is disposed in the gap **216** between the rotor **214** and the stator **212** or inner body portion **223c** of the planet carrier **223** that supports that the stator **212**. As illustrated in FIG. 3, the lubricant **218** may also be disposed radially between the wheel rim **225** and the inner body portion **223c** or the stator **212** (whichever is disposed radially adjacent the wheel rim **225**) in an outer gap **216a**. The lubricant **218** may be delivered to the outer gap **216a** via the channel **224** extending through the stator **212** and the inner body portion **223c**, or the lubricant **218** may reach the outer gap **216a** by traveling through other fluid channels defined within the lubricant supported electric motor **210**.

[0039] The system **200** may include a sealing portion **227** that extends between the outer portion **223a** of the planet carrier **223** and the ring gear **R**, with the sealing portion **227** being fixed to either the outer body portion **223a** or the ring gear **R**. The sealing portion **227** operates to seal the interior of the lubricant supported electric motor **210** and retain the lubricant **218** therein, such that the lubricant will be limited to exiting the lubricant supported electric motor **210** via a drain channel. The system **200** may also include wiring channels or passageways **230** extending through the pins **223b** and the inner body portion **223c** of the planet carrier **223** for receiving and routing wiring or the like to transfer current to the stator **212**.

[0040] As described above, the rotor **214** is disposed within the stator **212** and is supported by the lubricant **218**. The rotor **214** may therefore rotate relative to the stator **212** in response to providing the current to the stator **212**. The rotor **214** is fixedly attached to the sun gears **S**, which have outer teeth configured to engage the outer teeth of the planet gears **P**. When the rotor **214** rotates, the sun gears **S** accordingly rotate along with the rotor **214**. Rotation of the sun gears **S** causes a rotation of the planet gears **P** about their individual axes, which in turn cause a rotation of the ring gear **R** and the wheel rim **225**.

[0041] Thus, the wheel rim **225** is directly supported by the reduction mechanism **222**, in particular the ring gear **R**, and the reduction mechanism is directly supported by the lubricant supported electric motor **210** and attached to the

lubricant supported electric motor **210**. In particular, the stator **212** is supported by or supports the inner body portion **223c** of the planet carrier **223**. The system **200** can therefore provide integrated gear reduction, allowing for a smaller assembly, and eliminating additional connective components that would otherwise connect an electric motor with a reduction mechanism that is further attached to a wheel.

[0042] The foregoing description of the embodiments has been provided for purposes of illustration and description. It is not intended to be exhaustive or to limit the disclosure. Individual elements or features of a particular embodiment are generally not limited to that particular embodiment, but, where applicable, are interchangeable and can be used in a selected embodiment, even if not specifically shown or described. The same may also be varied in many ways. Such variations are not to be regarded as a departure from the disclosure, and all such modifications are intended to be included within the scope of the disclosure.

What is claimed is:

1. A lubricant supported electric motor comprising:
 - a stator extending along an axis;
 - a rotor extending along the axis and rotatably disposed around the stator in radially surrounding and spaced relationship to define at least one support chamber extending between the stator and rotor;
 - a lubricant disposed in the at least one support chamber for supporting the rotor around the stator; and
 - a wheel rim fixedly attached to the rotor and disposed in surrounding relationship with the rotor and the stator, wherein rotation of the rotor is directly transferred to the wheel rim such that the wheel rim rotates in accordance with the rotation of the rotor.
2. A lubricant supported electric motor as set forth in claim 1, wherein the stator is fixedly attached to an axle extending along the axis and configured for attachment to a vehicle chassis.
3. A lubricant supported electric motor as set forth in claim 2, further comprising:
 - a casing fixedly attached to the rotor for rotation therewith, the casing extending axially from the rotor in surrounding relationship with the stator and the axle to define an interior cavity disposed therebetween; and
 - the interior cavity disposed in fluid communication with the at least one support chamber.
4. A lubricant supported electric motor as set forth in claim 3, further comprising a seal extending between the casing and the axle for retaining the lubricant within the interior cavity during rotation of the rotor and the casing.
5. A lubricant supported electric motor as set forth in claim 3, wherein the stator defines a passageway disposed in fluid communication with the at least one support chamber for introducing the lubricant.
6. A lubricant supported electric motor as set forth in claim 5, further comprising:
 - the axle defining a fluid channel disposed in fluid communication with the passageway; and
 - a pump disposed in fluid communication with the channel for pumping the lubricant serially through the fluid channel and the passageway to the at least one support chamber.
7. A lubricant supported electric motor as set forth in claim 6, wherein the axle defines an outlet disposed in fluid communication with the internal cavity and the pump for

draining the lubricant from the at least one support chamber and the internal cavity back towards the pump.

8. A lubricant supported electric motor as set forth in claim 2, wherein the axle defines a wiring channel disposed in communication with the stator for transferring current to the stator.

9. A lubricant supported electric motor as set forth in claim 6, wherein the pump includes a thermal control mechanism for cooling or heating the lubricant pumped to the support chamber to a predetermined temperature.

10. A lubricant supported electric motor comprising:

a stator extending along an axis;

a rotor extending along the axis and rotatably disposed radially within the stator in spaced relationship to define at least one support chamber extending between the stator and rotor;

a lubricant disposed in the at least one support chamber for supporting the rotor within the stator;

a wheel rim disposed in radially surrounding relationship with the stator and the rotor; and

a planetary gear reduction mechanism operably interconnected to the rotor, the stator, and the wheel rim and configured to rotate the wheel rim in response to rotation of the rotor within the stator.

11. A lubricant supported motor as set forth in claim 10, wherein the planetary gear reduction mechanism includes a sun gear disposed along the axis, a plurality of a planet gears engaged with and circumferentially disposed around the sun gear, and a ring gear engaged with and circumferentially disposed around the planet gears; and wherein the rotor is interconnected with the sun gear, the stator is interconnected with the plurality of planet gears, and the wheel rim is interconnected with the ring gear to establish the operable interconnection of the planetary gear reduction mechanism.

12. A lubricant supported electric motor as set forth in claim 11, further comprising:

a planet carrier interconnected to the stator and defining a plurality of pins disposed in circumferentially fixed relationship about the axis, and

each of the plurality of planet gears rotatably supported by one of the plurality of pins to establish the operable connection between the stator and the plurality of planet gears.

13. A lubricant supported electric motor as set forth in claim 12, wherein the planet carrier includes an inner body portion disposed within the wheel rim in axially aligned relationship with the rotor and the stator and an outer body portion disposed axially outside of and in spaced relationship with the inner body portion, and each of the plurality of pins extending between the inner and outer body portions.

14. A lubricant supported electric motor as set forth in claim 13, wherein the inner body portion is disposed radially between the stator and the wheel rim.

15. A lubricant supported electric motor as set forth in claim 13, wherein the outer body portion is fixedly attached to a vehicle chassis.

16. A lubricant supported electric motor as set forth in claim 13, where the stator defines at least one passageway disposed in fluid communication with the at least one support chamber, the planet carrier defines at least one channel extending through the plurality of pins and disposed in fluid communication with the at least one passageway, and a pump disposed in fluid communication with the at least one channel for pumping the lubricant serially through the at least one channel and the at least one passageway to the support chamber.

17. A lubricant supported electric motor as set forth in claim 16, further comprising a seal extending between the ring gear and the outer body portions of the planet carrier for retaining the lubricant within the support chamber.

18. A lubricant supported electric motor as set forth in claim 16, wherein the inner body portion of the planet carrier is disposed in spaced relationship with the wheel rim to define an outer gap, and the at least one passageway of the stator is disposed in fluid communication with the outer gap to deliver the lubricant to the outer gap in addition to the support chamber.

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