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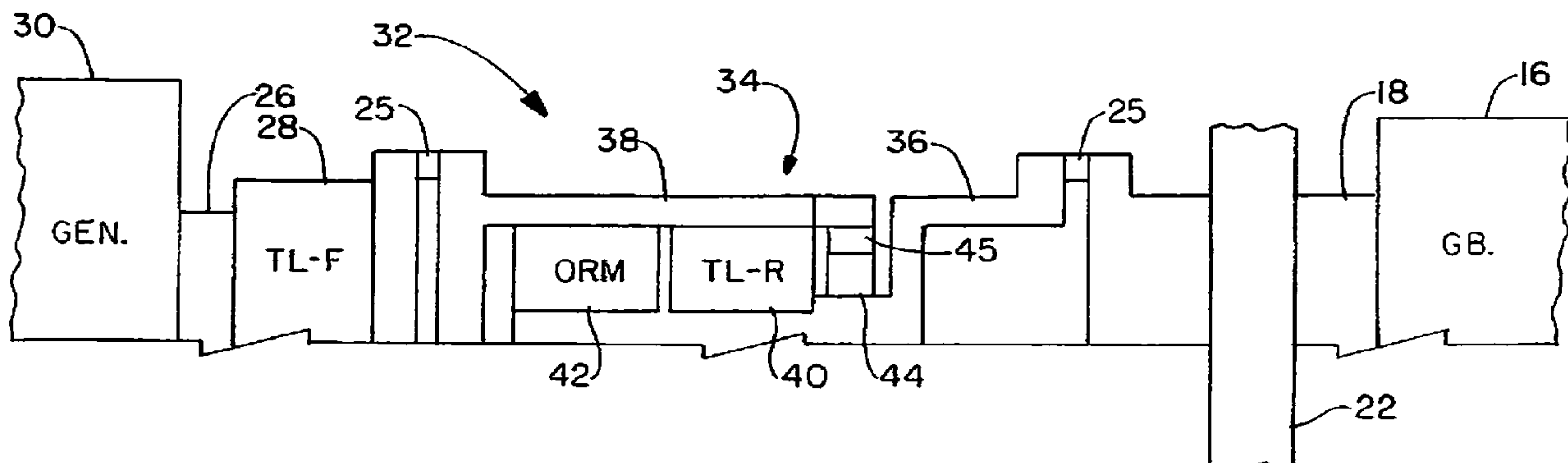
(72) **Inventeurs/Inventors:**
HEIDENREICH, DAVID, US;
OLSON, ERIK, US;
CULLINGS, GREGG, US;
PETERMAN, DAVID, US

(73) **Propriétaire/Owner:**
EBO GROUP, INC., US

(74) **Agent:** SIM & MCBURNEY

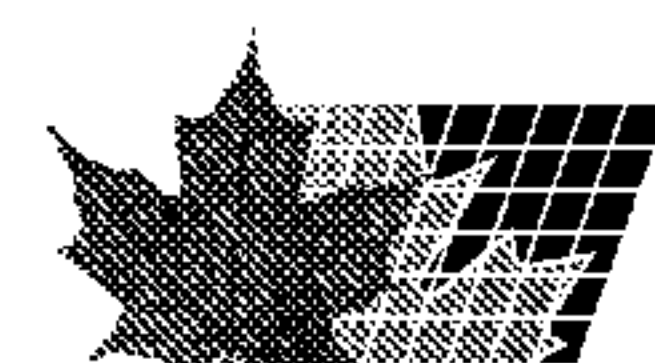
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(57) **Abrégé/Abstract:**

A wind turbine power generating system includes a wind turbine connected to a speed increasing gearbox having an output shaft. An electrical generator having an input shaft is also provided. An asymmetrical coupling interconnects the input and output shafts. The coupling includes a first torque limiting clutch in series connection with a one-way clutch between the input and output shafts. A second torque limiting clutch is provided in parallel connection with the one-way clutch between the input and output shafts. The first and second torque limiting clutches have different characteristic slip torques. The interconnection of the clutches, and the implementation of the one-way clutch, provides for effective operation of the first and second torque limiting clutches in opposite rotational directions of the input and output shafts. The second torque limiting clutch has a characteristic slip torque that is a fraction of the first torque limiting clutch, the second torque limiting clutch providing protection against reverse impulse loads.



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(72) Inventors; and

(75) Inventors/Applicants (for US only): **HEIDENREICH, David** [US/US]; 1916 Canterbury Road, Akron, Ohio 44333 (US). **OLSON, Erik** [US/US]; 3930 Minor Road, Akron, Ohio 44321 (US). **CULLINGS, Gregg** [US/US]; 1191 Ty Drive, Medina, Ohio 44256 (US). **PETERMAN, David** [US/US]; 2779 Jumpers Drive, Akron, Ohio 44333 (US).(74) Agents: **WEBER, Mark L.** et al.; Renner, Kenner, Greive, Bobak, Taylor & Weber, First National Tower -

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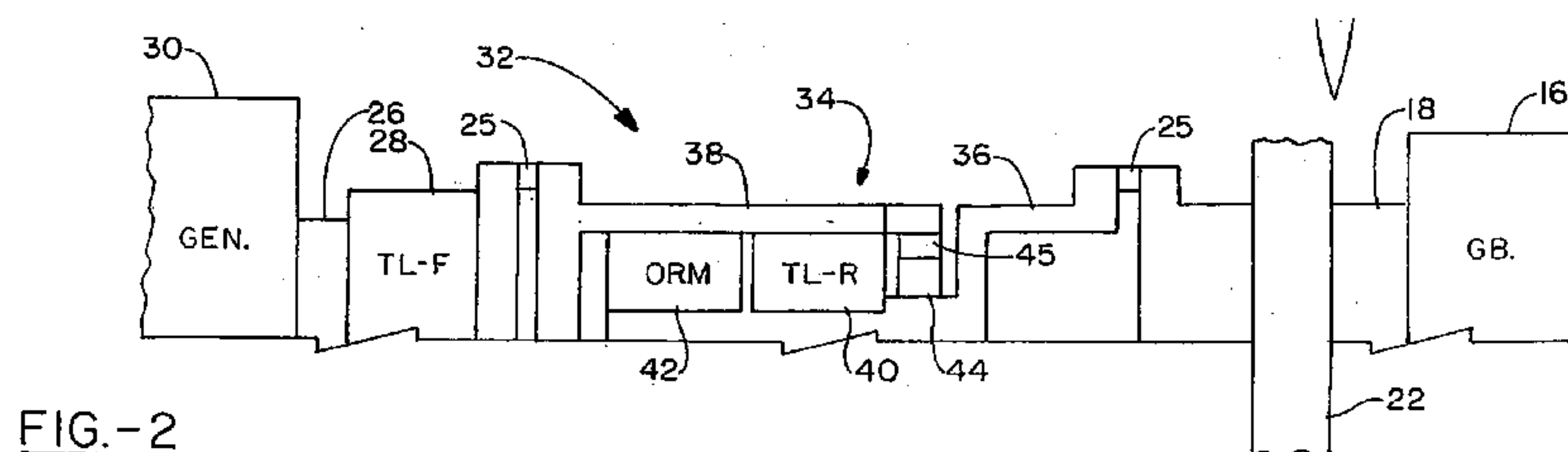


FIG. -2

(57) **Abstract:** A wind turbine power generating system includes a wind turbine connected to a speed increasing gearbox having an output shaft. An electrical generator having an input shaft is also provided. An asymmetrical coupling interconnects the input and output shafts. The coupling includes a first torque limiting clutch in series connection with a one-way clutch between the input and output shafts. A second torque limiting clutch is provided in parallel connection with the one-way clutch between the input and output shafts. The first and second torque limiting clutches have different characteristic slip torques. The interconnection of the clutches, and the implementation of the one-way clutch, provides for effective operation of the first and second torque limiting clutches in opposite rotational directions of the input and output shafts. The second torque limiting clutch has a characteristic slip torque that is a fraction of the first torque limiting clutch, the second torque limiting clutch providing protection against reverse impulse loads.

WIND TURBINE TORQUE LIMITING CLUTCH SYSTEM

TECHNICAL FIELD

The invention herein resides in the art of power transmission devices and, more particularly, to torque limiting clutches for use in wind turbines. Specifically, the invention relates to a torque limiting clutch for protecting wind turbines from bearing and gear damage caused by torque reversals. The invention provides an asymmetric torque limiting coupling system for use on wind turbines, in which a forward torque limiting clutch and a reverse torque limiting clutch are provide in paired relation, with the reverse torque limiting clutch having a characteristic slip torque that is a fraction of that of the forward torque limiting clutch. The invention contemplates the provision of such an arrangement of torque limiting clutches as a retrofit to existing systems, or as original equipment.

BACKGROUND ART

Tens of thousands of wind turbines have been installed over the past decade, almost all using a similar drive system incorporating a gearbox as a speed increaser, positioned between the turbine blades and a generator. Gearboxes are typically designed with the intent and desire to ensure that the bearings and gears are suitably aligned to take their intended design loads. Those design loads are typically focused on the forward operating rotational direction. However, it has been found that high reverse torques can affect the life of the bearings and gears of the gearbox. During torque reversals, the gears and bearings become misaligned, causing highly concentrated loading on the contact surfaces. Even a moderate reverse torque spike can damage the misaligned bearings and gears. There are various operating conditions that induce high torsional vibrations in the drive train and gearbox, some of which can cause severe torque in the reverse drive direction. These conditions may arise (a) upon start up when the electric contactor engages the wind turbine generator to the grid; (b) during emergency braking; (c) during normal braking when the caliper brake engages; (d) during

grid disconnect events; and (e) during any of various electrical faults and control malfunctions.

While gearboxes for wind turbines are typically designed for a 20 year bearing and gear life, it has been found that the average life of gearboxes in wind turbine designs is on the order of 7-11 years. The cost of gearbox replacement is extremely high, not only in direct costs, but in downtime, as well. Indeed, it is believed that premature gearbox failure in many wind turbine designs has been largely a consequence of reverse torque load, for which no effective protection has been provided. Many wind turbines have traditional friction torque limiting couplings, typically set at 150 - 200% of the rated torque of the wind turbine. These do not provide adequate protection for misaligned bearings and gears loaded in reverse. The present invention contemplates that an asymmetric torque limiter coupling system, with a high torque setting in the standard forward direction and a low torque setting in the reverse direction could significantly improve gearbox life.

DISCLOSURE OF INVENTION

In light of the foregoing, it is a first aspect of the invention to provide a wind turbine torque limiting clutch system in which gearbox protection is provide in both forward and reverse operating directions.

Another aspect of the invention is the provision of a wind turbine torque limiting clutch system in which bidirectional protection of an asymmetric nature is provided.

Yet a further aspect of the invention is the provision of a wind turbine torque limiting clutch system in which a reverse torque limiter is provided having a characteristic slip torque that is a fraction of that in the forward direction.

Still a further aspect of the invention is the provision of a wind turbine torque limiting clutch system in which the clutch mechanisms are enclosed, sealed and dry, and operative in an inert gas atmosphere.

Still another aspect of the invention is the provision of a wind turbine

torque limiting clutch system including a system for monitoring the phase angle and slip between the blade shaft and generator shaft to assess the torques incident to the gearbox.

Yet a further aspect of the invention is the provision of a wind turbine torque limiting clutch system that is readily adaptable to existing wind turbines for enhanced operation and durability.

The foregoing and other aspect of the invention that will become apparent as the detailed description proceeds are achieved by the improvement in a wind turbine power generating system comprising a wind turbine connected to a speed increasing gearbox having a high speed output shaft, and an electrical generator having an input shaft, the improvement comprising: a coupling system interconnecting said output and input shafts, said coupling system being asymmetrical, having a first characteristic slip torque in a first forward rotational drive direction, and a second characteristic slip torque in a second reverse rotational drive direction.

Other aspects of the invention are attained by a wind turbine power generating system, comprising: a wind turbine connected to a speed increasing gearbox having a high speed output shaft; an electrical generator having an input shaft; and an asymmetrical coupling interconnecting said input and output shafts; said asymmetrical coupling comprising a first torque limiting clutch in series connection with a one-way clutch between said input and output shafts, and a second torque limiting clutch in parallel connection with said one-way clutch between said input and output shafts, said first and second torque limiting clutches having different characteristic slip torques.

25

DESCRIPTION OF DRAWINGS

For a complete understanding of the various aspects and structures of the invention, reference should be made to the following detailed description and accompanying drawings wherein:

Fig. 1 is a functional schematic of a wind turbine generator according to

30

the prior art;

Fig. 2 is a functional schematic of a first embodiment of the invention, particularly adapted as a retrofit to existing coupling systems that have a traditional torque limiter;

5 Fig. 3 is a functional schematic of a retrofit for wind turbine generators that previously included no torque limiter; and

Fig. 4 is a functional schematic of a wind turbine generator having an asymmetrical torque limiting coupling according to the invention, and implemented as original equipment.

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DETAILED DESCRIPTION OF PREFERRED EMBODIMENT

Referring now to the drawings and more particularly to Fig. 1, it can be seen that a wind turbine generating system made in accordance with the prior art is designated generally by the numeral 10. The system 10 includes a series of
15 turbine blades 12 attached to a low speed turbine shaft 14. The turbine shaft 14 passes to a gearbox 16, which serves to greatly increase the rotational speed of the gearbox output shaft 18, in standard fashion. A caliper brake 20 is operative in association with a brake rotor 22, affixed to the gearbox shaft 18, to selectively stop rotation of the shaft 18, as readily appreciated by those skilled in the art. A
20 coupling is interposed between the gearbox output shaft 18 and a generator shaft 26, as shown. Those skilled in the art will appreciate that the coupling comprises a coupling spacer 24 with flexing elements 25 adapted to accommodate misalignment between the shafts 18, 26.

A torque limiter or torque limiting clutch 28 is provided in association with
25 the generator shaft 26 to isolate the gearbox output shaft 18 and connected gearbox 16 from excessive intermittent torque. Those skilled in the art will appreciate that the torque limiting clutch 28 will typically have a characteristic slip torque set at 1.2 to 2 times the nominal torque contemplated for application to the gearbox 16. In general, the torque limiting clutch 28 is set at 1.5 times
30 such nominal torque. Those skilled in the art will appreciate that the torque

limiting clutch 28 is typically bidirectional, operating with equal slip torque in both a clockwise and counterclockwise rotational mode. However, the characteristic slip torque limiter 28 is typically well in excess of that required for protection against reverse torque excursions.

5 As presented above, the design of the prior art wind turbine generating system 10, and associated torque limiting clutch 28, gave little thought to the presence of torque loads on misaligned bearings and gears in the reverse drive direction, such torque giving rise to gearbox damage and the necessity of early service and repair. The instant invention, shown in Figs. 2-4 and described
10 below, serves to remedy this problem by providing enhanced torque limiting protection against reverse loads.

With reference now to Fig. 2, it can be seen that a first embodiment of a wind turbine generating system made in accordance with the invention is designated generally by the numeral 32. This first embodiment is presented as
15 a retrofit for a system presently employing a torque limiting clutch. As shown, a gearbox 16 has an output drive shaft 18 in operative communication with a caliper brake 20 and brake rotor 22. The gearbox shaft 18 is interconnected with the generator shaft 26 by means of a coupling spacer 34 and flexing elements 25. The coupling spacer 34 includes a pair of concentric overlapping hubs 36, 38,
20 respectively interconnected with the shafts 18, 26. A first torque limiting clutch 28, typically having a slip torque of 1.2 to 2 times the nominal torque for the system 32 is interposed between the coupling spacer 34 and the generator shaft 26. The torque limiting clutch 28 is, in the embodiment of Fig. 2, a portion of the original equipment of the wind turbine generating system. To provide
25 protection for reverse torque occurrences, the original coupling between the shafts 18, 26 is replaced with the coupling spacer 34 comprising hubs 36, 38 with a torque limiting clutch 40 in parallel with an over running mechanism 42. The overrunning mechanism 42 will typically comprise a one-way clutch, providing for direct interconnection between the hubs 36, 38 in a first rotational direction,
30 and being freewheeling in the opposite direction. Accordingly, the torque limiting

clutch 40 serves as a reverse torque limiting clutch, providing for interconnection between the hubs 36, 38 with a characteristic slip torque in the reverse rotational direction. In a preferred embodiment of the invention, the torque limiting clutch 40 has a characteristic slip torque that is a fraction of the nominal torque experienced by the torque limiting clutch 28.

As further shown in Fig. 2, seal 44 may be employed to provide a sealed enclosure and protect the coupling spacer 34, and particularly the torque limiting clutch 40 and the over running mechanism 42 from the environment. A dry environment is preferred. To further protect these elements from the environment, it is contemplated that the enclosure is slightly pressurized with an inert gas such as nitrogen. The presence of a pressurized interior may be indicated by an appropriate pressure indicator 45, such as a bladder or spring-biased rod.

In the normal forward drive direction, the wind turbine generating system 32 of Fig. 2 operates such that the gearbox shaft 18 drives the generator shaft 26 through the coupling spacer 34 in a first rotational direction. Protection is provided by means of the torque limiting clutch 28, interposed between the coupling spacer 34 and the generator shaft 26. In this direction of rotation, the overrunning mechanism or one-way clutch 42 provides direct interconnection between the hubs 36, 38.

In the event of a reverse torque situation, the one-way clutch 42 becomes freewheeling and the interconnection between the hubs 36, 38 is provided through the reverse torque limiting clutch 40. The characteristic slip torque of the torque limiting clutch 40 is less than that of the torque limiting clutch 28 and, accordingly, protection to the gearbox 16 for reverse load torques is provided by the reverse torque limiting clutch 40.

With reference now to Fig. 3, it can be seen that a second embodiment of a wind turbine generating system is designated generally by the numeral 46. The system 46 again provides a retrofit for an existing system by replacing the original coupling spacer with the coupling spacer 48. This modification is particularly

adapted for systems that had no torque limiting clutch provision. Accordingly, the coupling spacer 48, provided between the gearbox shaft 18 and generator shaft 26 includes a pair of hubs 50, 52 with torque limiting interconnections interposed therebetween. Again, an overrunning mechanism or one-way clutch 54 is
5 provided in series connection with the torque limiting clutch 56, rendering the torque limiter 56 a forward torque limiting clutch, with the one-way clutch 54 being freewheeling in the opposite direction. A reverse torque limiting clutch 58 is interposed between the hubs 50, 52 in parallel with the interconnection between the one-way clutch 54 and forward torque limiting clutch 56. Again, the
10 characteristic slip torque of the torque limiting clutch 58 is but a fraction of that of the torque limiting clutch 56. Accordingly, the one-way clutch 54 ensures that the torque limiting clutch 56 controls forward torque applications, while the torque limiting clutch 58 controls reverse torque incidents.

The instant invention further contemplates the utilization of monitoring
15 and recording apparatus for assessing slip and torque between the various elements of the wind turbine generating system. In the embodiment of Fig. 3, variable reluctance magnetic sensors or transducers 60, 62, 64 are positioned in association with the gearbox shaft 18 (transducer 60), hub 50 (transducer 62), and generator shaft 26 (transducer 64). Each of the transducers 60, 62, 64
20 provides an output corresponding to rotational movement of the associated element 18, 50, 26. Accordingly, the slip and torque experienced along the power transmission drive 18, 48, 26 may be continually monitored by means of the monitor/recorder 66. As shown, the output of the generator 30 may also be so recorded.

25 Those skilled in the art will appreciate that not only the rotational speed of the various elements may be measured, but also the phase angle between such elements. For example, those skilled in the art will appreciate that the phase angle difference between the transducer 60 and the transducer 62 is an indication of the characteristic torsional wind up between the transducers, thus translatable
30 to the torque in the coupling system. When the transducers 60, 62 provide

signals that are in phase-lock correspondence with each other, there is no torque imparted to the shaft 18. Similarly, the differences in rotational speed monitored between the transducers 60 and 64 or 62 and 64 are indicative of any instantaneous slip between the elements with which the transducers are associated. The monitor/recorder 66 can thus obtain data regarding the torque and slip characteristics of the transmission shafts and their coupling in the wind turbine generating system 46.

With reference now to Fig. 4, it can be seen that an original equipment wind turbine generating system made in accordance with the invention is designated generally by the numeral 70. Here again, a gearbox 16 has an output shaft 18 interconnected through the coupling spacer 24 and flexing elements 25 to a hub 72 which is in operative driving communication with the generator shaft 26 of the generator 30. In an original equipment embodiment of the invention, a torque limiter 74, serving as a forward torque limiter, is interconnected in series with a one-way clutch or overrunning mechanism 76 to interconnect the hub 72 with the generator shaft 26. In parallel with this arrangement, and further interconnecting the hub 72 with the generator shaft 26, is a torque limiting clutch 78 which, as with the embodiment of Fig. 3, operates to provide torque limiting capabilities for oppositely directed rotation from that of the torque limiting clutch 74. This reverse torque limiter 78 has a characteristic slip torque that is a fraction of the that of the forward torque limiting clutch 74. Again, the operation is the same, as discussed above.

It is contemplated that the various embodiments of the invention will typically have a forward operating torque limiting clutch having a characteristic slip torque of 1.2 to 2 of the nominal torque experienced by the system, and most preferably on the order of 1.5 times such torque. The reverse torque limiting clutch will typically have a characteristic slip torque less than that of the forward torque limiting clutch. It is further contemplated that the torque limiting clutch of the invention may include multiple disk clutches, as are well known and understood by those skilled in the art. Of course, the size and number of such

disks will typically be determined by reactionary force needs, with due consideration being given to packaging constraints. As presented above, the invention further contemplates that the torque limiting clutches of the invention, as well as the over running mechanism will typically be housed in an inert gas
5 environment, with the inert gas being under pressure greater than that of atmosphere, ensuring protection from the environment. Indeed, according to a preferred embodiment of the invention, a nitrogen filled housing is contemplated.

The monitor/recorder 26 of the invention is contemplated for use in recording and logging excessive torque and/or torque slip events. The
10 transducers 60, 62, 64 may be of any suitable nature, but are presently contemplated as being variable reluctance metallic ring transducers with spaced teeth inducing signals corresponding to rotational speed and position. Signal differences are an indication of torque and/or slippage, as will be appreciated by the skilled artisan.

15 The instant invention, in contradistinction to the prior art, seeks to protect the gearbox of wind turbine generating systems from the harmful effects of the reverse torques periodically impacting the gearbox of such systems. By providing the asymmetrical torque limiting couplings presented and described above, with the forward and reverse torque limiters being independently operative and of
20 different settings, protection of gearboxes is ensured.

Thus it can be seen that the various aspects of the invention have been satisfied by the structure presented above. While in accordance with the patent statutes only the best mode and preferred embodiments of the invention have been presented and described in detail, the invention is not limited thereto or
25 thereby. Accordingly, for an appreciation of the scope and breadth of the invention reference should be made to the following claims.

What is claimed is:

1. A wind turbine power generating system comprising:
a wind turbine connected to a speed increasing gearbox having a high speed output shaft;
an electrical generator having an input shaft; and
a coupling system interconnecting said output and input shafts, said coupling system being asymmetrical, having a first characteristic slip torque in a first forward rotational drive direction, and a second characteristic slip torque in a second reverse rotational drive direction.
2. The wind turbine power generating system according to claim 1, wherein said first characteristic slip torque is greater than a nominal torque rating of the wind turbine, and said second characteristic slip torque is less than the nominal torque rating of the wind turbine.
3. The wind turbine power generating system according to claim 2, wherein said coupling system comprises a first torque limiting clutch having said first characteristic slip torque, and a second torque limiting clutch having said second characteristic slip torque.
4. The wind turbine power generating system according to claim 3, further comprising a one-way clutch between said output and input shafts, said one-way clutch being in engagement in said first forward rotational drive direction, and free-wheeling in said second reverse rotational drive direction.
5. The wind turbine power generating system according to claim 4, wherein said first torque limiting clutch is in series connection with said one-way clutch between said output and input shafts.
6. The wind turbine power generating system according to claim 5, wherein said one-way clutch is in parallel connection with said second torque limiting clutch.

7. The wind turbine power generating system according to claim 6, wherein said coupling system comprises concentric hubs, with said one-way clutch and said second torque limiting clutch in parallel interconnection therebetween.
8. The wind turbine power generating system according to claim 6, wherein said coupling system comprises a hub about one of said output and input shafts, and wherein said first and second torque limiting clutches and said one-way clutch are interposed between said hub and one of said output and input shafts.
9. The wind turbine power generating system according to any one of claims 6 to 8, wherein said first and second torque limiting clutches and said one-way clutch each comprise at least one friction drive disk.
10. The wind turbine power generating system according to any one of claims 6 to 9, wherein at least one of said first and second torque limiting clutches and said one-way clutch are maintained in a sealed enclosure.
11. The wind turbine power generating system according to claim 10, wherein said sealed enclosure is filled with an inert gas.
12. The wind turbine power generating system according to claim 11, wherein said inert gas is maintained in said sealed enclosure under pressure.
13. The wind turbine power generating system according to claim 12, further comprising a pressure indicator providing an indicia of said pressure within said sealed enclosure.
14. The wind turbine power generating system according to any one of claims 6 to 13, further comprising sensors interconnected among said output and input shafts and said coupling system and connected to a monitor to measure torque within said coupling system.

15. The wind turbine power generating system according to claim 14, wherein said sensors enable said monitor to measure slip in said first and second torque limiting clutches.

16. The wind turbine power generating system according to claim 15, wherein said sensors comprise variable reluctance metallic ring transducers.

17. A wind turbine power generating system, comprising:

a wind turbine connected to a speed increasing gearbox having a high speed output shaft;

an electrical generator having an input shaft; and

an asymmetrical torque limiting coupling interconnecting said input and output shafts, said asymmetrical torque limiting coupling comprising a first torque limiting clutch in series connection with a one-way clutch between said input and output shafts, and a second torque limiting clutch in parallel connection with said one-way clutch between said input and output shafts, said first and second torque limiting clutches having different characteristic slip torques.

18. The wind turbine power generating system according to claim 17, wherein said characteristic slip torque of said second torque limiting clutch is a fraction of said characteristic slip torque of said first torque limiting clutch.

19. The wind turbine power generating system according to claim 17 or 18, wherein said first and second torque limiting clutches have effective operation in opposite rotational directions.

20. The wind turbine power generating system according to any one of claims 17 to 19, wherein said first and second torque limiting clutches and said one-way clutch are dry and maintained in a sealed enclosure.

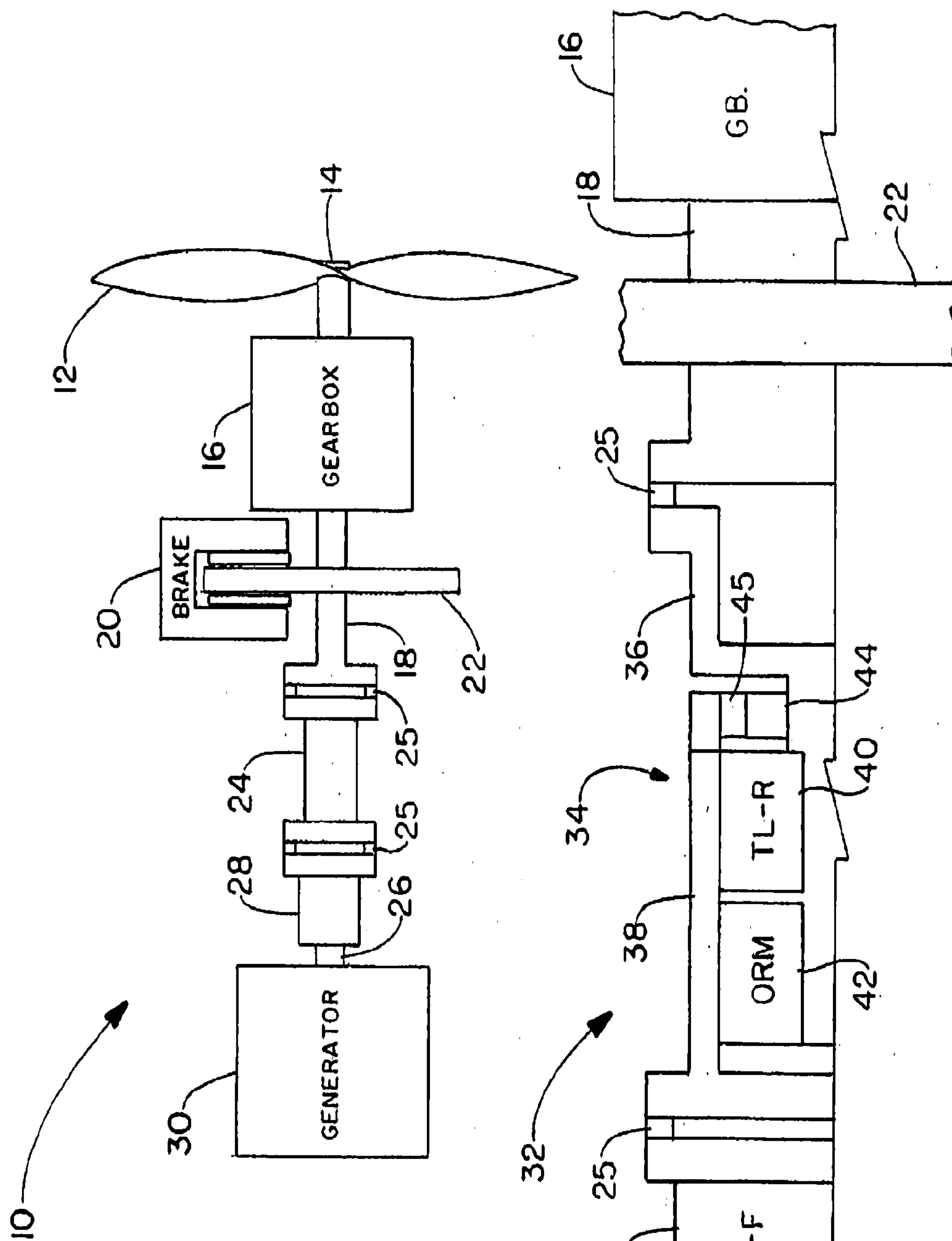


FIG. - 1

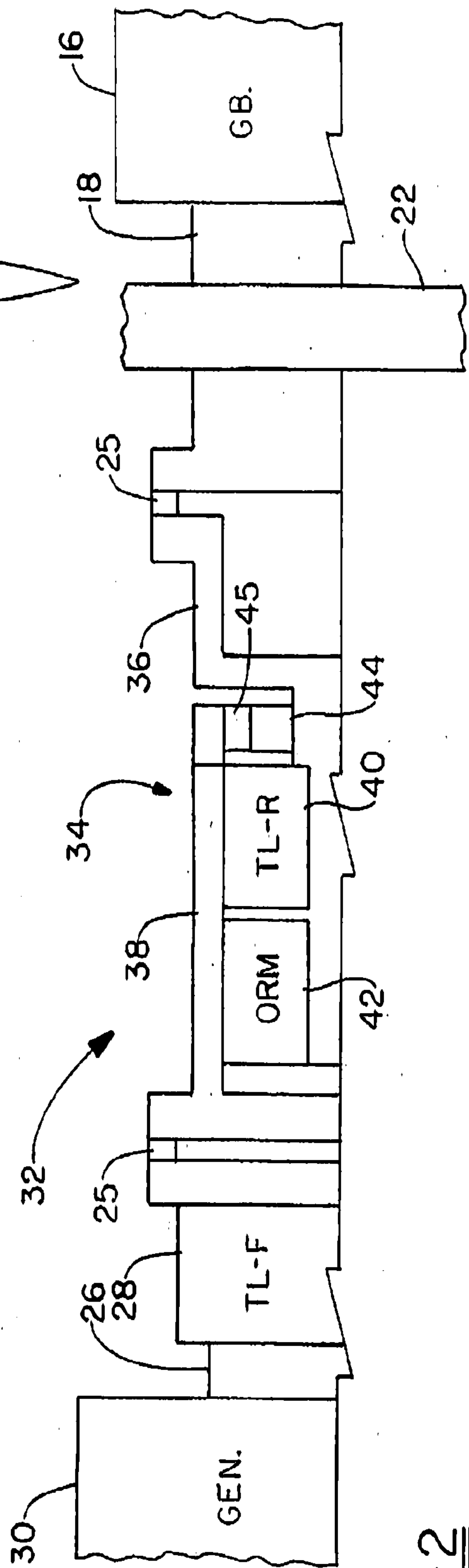


FIG.-2

