



(51) International Patent Classification:

G01L 5/13 (2006.01) G01L 3/24 (2006.01)
G01G 19/08 (2006.01) G01M 15/04 (2006.01)

(21) International Application Number:

PCT/SE2010/000299

(22) International Filing Date:

15 December 2010 (15.12.2010)

(25) Filing Language:

English

(26) Publication Language:

English

(71) Applicant (for all designated States except US): **VOLVO LASTVAGNAR AB** [SE/SE]; S-405 08 Göteborg (SE).

(72) Inventors; and

(75) Inventors/Applicants (for US only): **BRÄTHE, Lars** [SE/SE]; V. Stillestorpsgratan 21, S-417 13 GÖTEBORG (SE). **ERIKSSON, Anders** [SE/SE]; Soldatvägen 2, S-423 49 Torslanda (SE).

(74) Agent: **FRÖHLING, Werner**; Volvo Technology Corporation, Corporate Patents, 06820, M1.7, S-405 08 Göteborg (SE).

(81) Designated States (unless otherwise indicated, for every kind of national protection available): AE, AG, AL, AM, AO, AT, AU, AZ, BA, BB, BG, BH, BR, BW, BY, BZ, CA, CH, CL, CN, CO, CR, CU, CZ, DE, DK, DM, DO, DZ, EC, EE, EG, ES, FI, GB, GD, GE, GH, GM, GT, HN, HR, HU, ID, IL, IN, IS, JP, KE, KG, KM, KN, KP, KR, KZ, LA, LC, LK, LR, LS, LT, LU, LY, MA, MD, ME, MG, MK, MN, MW, MX, MY, MZ, NA, NG, NI, NO, NZ,

[Continued on next page]

(54) Title: TORQUE CALIBRATION METHOD

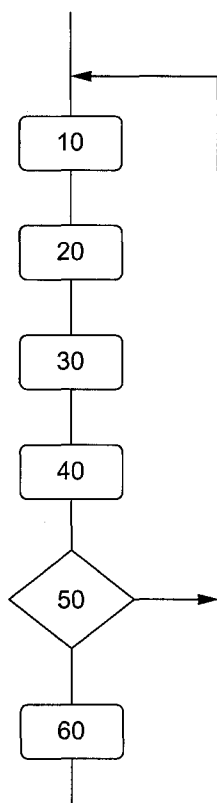


FIG. 1

(57) Abstract: Method for verifying an engine torque estimation, comprising the steps of estimating the engine torque based on the amount of fuel injected into the engine, where the engine torque is obtained from a fuel injection table, estimating a first vehicle weight value by a calculation based on acceleration of the vehicle and the estimated engine torque, estimating an auxiliary brake torque by using an auxiliary brake table, estimating a second vehicle weight value by a calculation based on the estimated auxiliary brake torque, and comparing the first vehicle weight value with the second vehicle weight value. The advantage of the invention is that it is possible to detect if an actual engine torque value deviates from the original engine torque value of a vehicle without measuring the engine torque with a separate torque sensor.

WO 2012/082019 A1



OM, PE, PG, PH, PL, PT, RO, RS, RU, SC, SD, SE, SG, SK, SL, SM, ST, SV, SY, TH, TJ, TM, TN, TR, TT, TZ, UA, UG, US, UZ, VC, VN, ZA, ZM, ZW.

EE, ES, FI, FR, GB, GR, HR, HU, IE, IS, IT, LT, LU, LV, MC, MK, MT, NL, NO, PL, PT, RO, RS, SE, SI, SK, SM, TR), OAPI (BF, BJ, CF, CG, CI, CM, GA, GN, GQ, GW, ML, MR, NE, SN, TD, TG).

(84) Designated States (unless otherwise indicated, for every kind of regional protection available): ARIPO (BW, GH, GM, KE, LR, LS, MW, MZ, NA, SD, SL, SZ, TZ, UG, ZM, ZW), Eurasian (AM, AZ, BY, KG, KZ, MD, RU, TJ, TM), European (AL, AT, BE, BG, CH, CY, CZ, DE, DK,

Published:

— with international search report (Art. 21(3))

TORQUE CALIBRATION METHOD

TECHNICAL FIELD

The present invention relates to a method for verifying a torque estimation of an engine. The inventive method is adapted for engines having an auxiliary brake. Thus, the method is well suited for the use in heavy vehicles.

BACKGROUND ART

In modern heavy vehicles, such as trucks, tractors, busses and other commercial vehicles, several systems for estimating and controlling the driving parameters of the engine and of the vehicle are known. These often relates to compensating the driving parameters of the engine and the braking parameters for the brake systems. Depending on the available sensors on the vehicle, some parameters can be measured directly by the sensors, and other parameters must be estimated by using the available measurements.

Some vehicles, e.g. the ones equipped with air suspension, have the capability to measure the weight of the load of the vehicle directly by using the air suspension pressure sensors. The total weight of the vehicle can thus easily be obtained since the unloaded weight of the vehicle is known. Other vehicles, e.g. such that have conventional leaf springs, are usually not provided with sensors that can measure the weight of the load directly. Instead, different ways of obtaining the weight of the vehicle are known.

In one method, an estimate of the weight of a vehicle is calculated from a measured acceleration and an estimated or measured engine torque. The engine torque can be measured by a torque sensor positioned e.g. on the gearbox input shaft. An estimate for the engine torque may be based on the amount of fuel injected into the engine. The amount of injected fuel is

input in a fuel injection table which will output an estimate for the engine torque at any given time. In this way, an estimate for the weight of the vehicle or the vehicle combination can be provided.

5 In order to be able to set weight dependent control parameters of the vehicle, it is important that the estimated weight of the vehicle is as precise as possible. If the estimated weight is lower than the actual weight, the vehicle may be overloaded when an additional load is loaded on the vehicle. It is also possible that the brakes are applied with a too low brake
10 pressure which may lead to a longer brake distance. If the estimated weight is higher than the actual weight, the fuel consumption may be higher than necessary and the vehicle may not be operated as economical as possible. For a vehicle that is not provided with an engine torque sensor, it is thus important that the engine torque is estimated properly.

15

US 6144928 describes a method for determining the weight of a vehicle, in particular for a commercial vehicle having a towing vehicle and a trailer/semitrailer. By the method, it is possible to determine the mass distribution relation between the complete vehicle combination and the
20 towing vehicle alone. This will allow for a proper distribution of the brake torque to individual wheel brakes, and between the towing vehicle and the trailer.

US 6633006 describes a method for determining the mass of a vehicle
25 with at least two measurements offset in time within a measuring period, where one measurement is performed during a traction-free phase and the other measurement is performed during a traction phase. One measurement is preferably performed within the traction-free phase during a gearshift and the other during a traction phase before or after the
30 gearshift.

US2010/00049415 describes a method for compensating a vehicle brake value based on the mass of the vehicle.

In these and other methods, the weight of the vehicle is estimated in
5 different ways. There is however still room for improvements.

DISCLOSURE OF INVENTION

An object of the invention is therefore to provide a method for verifying an engine torque estimation. A further object of the invention is to provide a
10 method for detecting manipulated engine control parameters. A further object of the invention is to provide a method for detecting a non-approved engine modification. Another object of the invention is to provide a method for ensuring that the vehicle can be driven safely after a non-approved modification is detected.

15 The solution to the problem according to the invention is described in the characterizing part of claim 1. The other claims contain advantageous further developments of the inventive method.

In a method for verifying an engine torque estimation, comprising the steps of estimating the engine torque based on the amount of fuel injected
20 into the engine, where the engine torque is obtained from a fuel injection table and estimating a first vehicle weight value by a calculation based on the acceleration of the vehicle and the estimated engine torque, the object of the invention is achieved in that an auxiliary brake torque is estimated by using an auxiliary brake table, a second vehicle weight value is
25 estimated by a calculation based on the estimated auxiliary brake torque, and in that the first vehicle weight value is compared with the second vehicle weight value.

By this first embodiment of the method for verifying an engine torque estimation according to the invention, it is possible to detect if the actual

engine torque of a vehicle deviates from the original engine torque of that vehicle. This is done without using a specific separate torque sensor. The advantage of this is that a manipulation of the control parameters for the engine in the engine control system can be detected. Such manipulation
5 can lead to excess wear of the engine, to overload of the engine or engine components and/or to excessive temperatures of the engine or engine components. Further, when the estimated engine torque is used to calculate the weight of the vehicle, the calculated weight may be erroneous which in turn may affect the weight dependent parameters of
10 the vehicle.

In an advantageous development of the inventive method, the auxiliary brake is an engine brake and the auxiliary brake table uses the engine speed as input. In this way, the engine torque can be obtained in an easy way and there is no need for a retarder or the like.

15 In an advantageous development of the inventive method, a message is provided which contains the difference between the first vehicle weight value and the second vehicle weight value. In this way, the result can be presented in an easy way. Such a message may be given to the driver of the vehicle or may be sent to a stationary control station. The message
20 may also be stored in a memory for further use.

In an advantageous development of the invention, the first vehicle weight values and the second vehicle weight values are stored in a table within predefined time intervals. In this way, it is possible to obtain a long term history for the vehicle. This makes it possible to detect if a vehicle is driven
25 with manipulated engine parameters at some periods and with the regular engine parameters when maintenance is due. It may also be possible to detect if a different fuel has been used at some time periods.

BRIEF DESCRIPTION OF DRAWINGS

The invention will be described in greater detail in the following, with reference to the attached drawing, in which

Fig. 1 shows a flow chart of an example of the method according to the invention.

5 MODES FOR CARRYING OUT THE INVENTION

The embodiments of the invention with further developments described in the following are to be regarded only as examples and are in no way to limit the scope of the protection provided by the patent claims.

In the method described herein, a truck will be used as an example of a
10 vehicle. Other types of heavy vehicles in which the inventive solution may be used are tractors with or without a trailer and busses. It is also possible to use the invention on smaller vehicles, such as distribution lorries and vans, as long as the engine is provided with an engine brake of some sort or that the vehicle is equipped with an auxiliary brake. An engine brake
15 can be e.g. an exhaust brake or a compression brake. An auxiliary brake can be e.g. a hydraulic retarder or an electromagnetic auxiliary brake.

The purpose of the inventive method is to verify that an engine torque estimation performed in a conventional way is correct. Since the weight estimation on vehicles that are not equipped with load sensors is
20 dependent on the engine torque estimation, it is important that the engine torque estimation is correct in order to be able to set weight dependent parameters of the vehicle to correct values. There are different reasons for the engine torque estimation to deviate from the actual engine torque. One obvious reason is that the parameters of the control electronics or the
25 hardware of the engine is modified in a non-approved way. Such a modification may e.g. be a replaced injection pump, a replaced inlet compressor or modified engine control parameters, so called chip tuning. A modification of the control parameters may take place in the engine control computer or may be a unit mounted after the engine control

computer. It is also possible to manipulate one or more sensors sending signals to the engine control computer in order to modify the control parameters. If these modifications are made by an external party and if the modifications are not approved by the manufacturer of the vehicle, the modifications may not be detected by the vehicle control system. The engine may thus deliver a higher engine torque than the original nominal engine torque at a given fuel injection set reference value. Another reason for the engine torque to differ is if a non-approved fuel is used.

A conventional engine torque estimation is done by reading the set reference value for the fuel injection or measure the amount of injected fuel at a given engine rotational speed. This reference value or measured value is used as input to a stored fuel injection table which gives an engine torque value as output. This engine torque value assumes a specific fuel quality. Since the engine torque is not measured directly, it is important that the engine behaviour can be predicted by using the fuel injection reference value.

The weight of the vehicle is calculated by using the estimated engine torque value together with a measured acceleration for a given time period. A simplified equation for this is force equals mass times acceleration. In this equation, further parameters can of course be included to obtain a higher accuracy, i.e. driveline efficiency, slope resistance, rolling resistance and/or wind resistance. The use of such additional parameters may be dependent on available sensors for that parameter. The slope resistance may e.g. be measured by an inclination sensor or the road profile may be obtained from a GPS based map system. The obtained vehicle weight value can be used to set different weight dependent vehicle parameters, such as braking parameters, and may also be used to deduct the total load on the vehicle. This can in turn be used for loading purposes in order to keep specific weight limitations.

If the engine torque value is not correct, the estimated vehicle weight will also not be correct. If the engine is manipulated with the purpose of giving a higher engine torque than the nominal engine torque, the estimated vehicle weight will be lower than the actual vehicle weight. Weight
5 dependent parameters, such as brake parameters, may thus be affected such that the brake distance may be longer than necessary. It may also mislead a driver to overload the vehicle. A further disadvantage of manipulated engine parameters is that the engine may be driven harder than intended at a given situation. This may lead to an overloaded engine,
10 excessive high temperatures in the engine and an overloaded driveline. This may affect the life of the vehicle in a negative way with higher maintenance costs for the owner and/or higher warranty costs for the manufacturer.

In the inventive method, an additional vehicle weight estimation based on
15 an auxiliary brake torque estimation is thus performed. The brake torque of an auxiliary brake is simple to determine. For an auxiliary brake mounted outside of the engine, the brake torque can be obtained from the rotational speed where the brake is mounted, e.g. by the rotational speed of the drive shaft when the brake is mounted after the gear box. For an
20 engine brake, the brake torque is mainly dependent on the engine speed and can thus be easily obtained. The actual brake torque of the auxiliary brake is also dependent on the requested brake pressure. An auxiliary brake, e.g. a retarder, may have a number of fixed brake levels. In this case, each brake level will have a specific brake torque table relating to
25 the rotational speed. If the requested brake torque is variable, the table will comprise a function relating the requested brake torque to the rotational speed.

Thus, the brake torque can be estimated in a reliable way without it being affected by a manipulated fuel injection. The engine speed or the
30 rotational speed of the auxiliary brake together with the requested brake

level is input to a stored auxiliary brake table which outputs the brake torque. The estimated weight is calculated from the brake torque in a similar way as the estimated engine torque as described above, by using the deceleration of the vehicle during a specific time period.

- 5 When measuring the deceleration of the vehicle, it is important that only the auxiliary brake is used during this measurement and not any wheel brakes. On several occasions, when the vehicle is coasting, this can be achieved without affecting the performance of the vehicle and without the attention of the driver. In practice, this is done such that when a smaller
- 10 brake torque is requested by the driver, e.g. when the driver pushes slightly on the brake pedal and the system detects that the required brake torque can be delivered by using only the auxiliary brake, an estimation is performed at the same time. An estimation can also be performed when a cruise control is used by the driver to hold a specific cruising speed.
- 15 The vehicle weight value resulting from the brake torque estimation can then be compared with the vehicle weight value resulting from the engine torque estimation. If the two vehicle weight values differ from each other by at least a predefined factor, it can be assumed that the engine is running under a non-approved condition. This may e.g. include
- 20 manipulated control parameters for the engine, replaced engine components or a non-approved fuel. Specifically, if the vehicle weight value resulting from the brake torque estimation is higher than the vehicle weight value resulting from the engine torque estimation by a predefined factor, it can be assumed that the engine control parameters have been
- 25 manipulated. The predefined factor used is dependent on different environmental and vehicle parameters, but should be higher than the tolerances for both the engine torque estimation and the brake torque estimation. A factor deviating more than 3% is plausible.

30 Since the vehicle weight values may vary some depending on environmental conditions, the vehicle weight values resulting from the

engine torque estimation and the vehicle weight value resulting from the brake torque estimation can be stored in a table. These values can be used to obtain an average vehicle weight value and can also be used to obtain a long term trend for the vehicle weight values and the deviation
5 between the vehicle weight values. In this way, it is possible to see deviations over time, which may e.g. show when a different fuel was used or when an engine parameter manipulation was performed.

If there is a deviation between the vehicle weight value resulting from the engine torque estimation and the vehicle weight value resulting from the
10 brake torque estimation, a message can be provided by the vehicle control system. The message can be stored in a memory and can be read during maintenance. A message can also be given to the driver, e.g. on a display or by a dedicated lamp or the like. It is also possible to send the message to a central stationary control station, either at the maintenance centre or
15 at the manufacturer, such that the message can be stored in the history for a specific vehicle.

It is further possible that the vehicle control system reduces the output power of the engine by a value based on the vehicle weight value resulting from the brake torque estimation. In this way, the stored fuel injection table
20 is adapted to a reversed calculated engine torque value obtained from the vehicle weight value resulting from the brake torque estimation. This will allow the vehicle to be used in a safe way without the risk of overloading the engine. It is also possible to reduce the output power of the engine by a larger amount, in order to ensure that there will be no overload of the
25 engine or the vehicle until the cause of the problem is found and corrected. Such a solution may be to reprogram the engine control system with the proper software.

Fig. 1 shows a flow chart of an example of the inventive method. In step 10, the engine torque is estimated by using the amount of fuel injected into

the engine as input to a fuel injection table and from that table, an engine torque value is outputted.

In step 20, a first vehicle weight value is calculated based on the acceleration of the vehicle and the estimated engine torque value obtained
5 in step 10.

In step 30, the engine brake torque is estimated by using the engine speed as input to a brake torque table which gives an engine brake torque value as output.

In step 40, a second vehicle weight value is calculated based on the
10 engine brake torque and the deceleration of the vehicle.

In step 50, the first vehicle weight value is compared with the second vehicle weight value.

In step 60, a message is provided if the difference between the first vehicle weight value and the second vehicle weight value differs with more
15 than a predefined value. The message may be stored in a memory, it may be given to the driver or it may be sent to a stationary control station. When a message has been provided, the system may continue with a new estimation, preferably after a specific time interval.

If the difference between the two vehicle weight values is below the
20 predefined value, no message is created and the method may continue with a new estimation, preferably after a specific time interval.

The invention is not to be regarded as being limited to the embodiments described above, a number of additional variants and modifications being possible within the scope of the subsequent patent claims.

CLAIMS

1. Method for verifying an engine torque estimation, comprising the steps of:
 - 5 - estimating the engine torque based on the amount of fuel injected into the engine, where the engine torque is obtained from a fuel injection table,
 - estimating a first vehicle weight value by a calculation based on acceleration of the vehicle and the estimated engine torque,
 - 10 characterized in that the method further comprises the steps of:
 - estimating an auxiliary brake torque by using an auxiliary brake table,
 - estimating a second vehicle weight value by a calculation based on the estimated auxiliary brake torque,
 - 15 - comparing the first vehicle weight value with the second vehicle weight value.
2. Method according to claim 1, characterized in that the auxiliary brake is an engine brake and that the auxiliary brake table uses the engine speed as input.
- 20 3. Method according to claim 1, characterized in that the auxiliary brake is an external retarder and that the auxiliary brake table uses the rotational speed of the auxiliary brake as input.
4. Method according to any of the preceding claims, characterized in that the method further comprises the
- 25

step of providing a message containing the difference between the first vehicle weight value and the second vehicle weight value.

- 5
5
5. Method according to any of the preceding claims, characterized in that the method further comprises the step of providing a message if the first vehicle weight value differs from the second vehicle weight value by at least a predefined value.
- 10
6. Method according to any of the preceding claims, characterized in that the method further comprises the step of providing a message when the second vehicle weight value is larger than the first vehicle weight value by a predefined value.
7. Method according to claim 5, characterized in that the predefined value is larger than 3%.
- 15
8. Method according to any of the preceding claims, characterized in that the first vehicle weight value and the second vehicle weight value are stored in a table at predefined time intervals.
- 20
9. Method according to claim 8, characterized in that the first vehicle weight value and the second vehicle weight value are stored in a table at predefined time intervals when the difference between the first vehicle weight value and the second vehicle weight value is above a predefined value.
- 25
10. Method according to any of claims 4 to 9, characterized in that the message is sent to a stationary control station.
11. Method according to any of claims 4 to 9, characterized in that the message is given to the driver of the vehicle by an optical and/or an audio signal.

12. Method according to any of any of the preceding claims,
characterized in that the method further comprises the
step of decreasing the power output of the engine if the difference
between the first vehicle weight value and the second vehicle
weight value is above a predefined value.
- 5

1 / 1

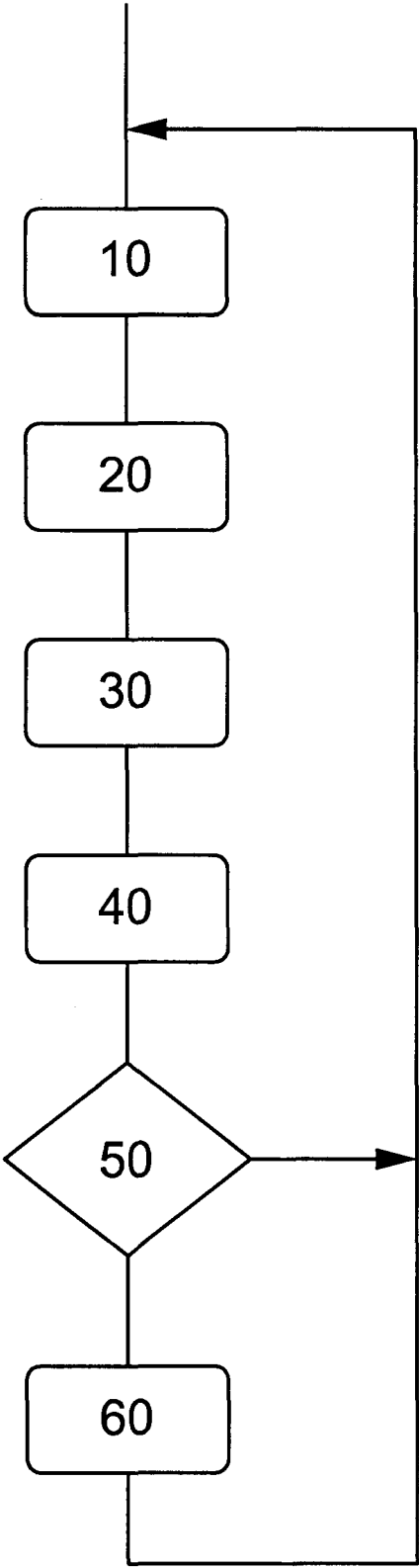


FIG. 1

INTERNATIONAL SEARCH REPORT

International application No.
PCT/SE2010/000299

A. CLASSIFICATION OF SUBJECT MATTER		
IPC: see extra sheet		
According to International Patent Classification (IPC) or to both national classification and IPC		
B. FIELDS SEARCHED		
Minimum documentation searched (classification system followed by classification symbols)		
IPC: G01G, G01L, G01M		
Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched		
SE, DK, FI, NO classes as above		
Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)		
EPO-Internal, PAJ, WPI data		
C. DOCUMENTS CONSIDERED TO BE RELEVANT		
Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 20040181317 A1 (FLECHTNER HORST ET AL), 16 September 2004 (2004-09-16); paragraphs [0045], [0047], [0052], [0053] --	1-12
X	WO 9318375 A1 (CATALYTIC IGNITER SYSTEMS ET AL), 16 September 1993 (1993-09-16); page 20, line 24 - line 34; page 21, line 7 - line 12; page 22, line 19 - line 21; page 22, line 26 - line 30 --	1-12
X	US 4656876 A (FREMD RAINER), 14 April 1987 (1987-04-14); column 1, line 42 - line 53 --	1-12
<input checked="" type="checkbox"/> Further documents are listed in the continuation of Box C. <input checked="" type="checkbox"/> See patent family annex.		
* Special categories of cited documents: "A" document defining the general state of the art which is not considered to be of particular relevance "E" earlier application or patent but published on or after the international filing date "L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified) "O" document referring to an oral disclosure, use, exhibition or other means "P" document published prior to the international filing date but later than the priority date claimed "T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention "X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone "Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art "&" document member of the same patent family		
Date of the actual completion of the international search		Date of mailing of the international search report
19-08-2011		19-08-2011
Name and mailing address of the ISA/SE Patent- och registreringsverket Box 5055 S-102 42 STOCKHOLM Facsimile No. + 46 8 666 02 86		Authorized officer Sture Elnäs Telephone No. + 46 8 782 25 00

INTERNATIONAL SEARCH REPORT

International application No.
PCT/SE2010/000299

C (Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT		
Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 4548079 A (KLATT ALFRED), 22 October 1985 (1985-10-22); column 1, line 64 - column 2, line 5; column 6, line 30 - line 32; column 6, line 47 - line 50 --	1-12
A	US 6185996 B1 (HE CHUAN ET AL), 13 February 2001 (2001-02-13); column 1, line 67 - column 2, line 19; column 10, line 57 - line 65 -- -----	1-12

Continuation of: second sheet

International Patent Classification (IPC)

G01L 5/13 (2006.01)

G01G 19/08 (2006.01)

G01L 3/24 (2006.01)

G01M 15/04 (2006.01)

Download your patent documents at www.prv.se

The cited patent documents can be downloaded:

- From "Cited documents" found under our online services at www.prv.se
(English version)
- From "Anförda dokument" found under "e-tjänster" at www.prv.se
(Swedish version)

Use the application number as username. The password is **MFTNFCSGZM**.

Paper copies can be ordered at a cost of 50 SEK per copy from PRV InterPat (telephone number 08-782 28 85).

Cited literature, if any, will be enclosed in paper form.

INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No.

PCT/SE2010/000299

US	20040181317 A1	16/09/2004	DE	10148091 A1	17/04/2003
			DE	50207127 D1	20/07/2006
			EP	1430276 A1	23/06/2004
			JP	4048176 B2	13/02/2008
			JP	2005504311 A	10/02/2005
			US	7363116 B2	22/04/2008
			WO	03029764 A1	10/04/2003
WO	9318375 A1	16/09/1993	NONE		
US	4656876 A	14/04/1987	DE	3429184 A1	13/02/1986
			FR	2569004 A1	14/02/1986
			GB	2162955 A	12/02/1986
			IT	1184650 B	28/10/1987
			JP	1842936 C	12/05/1994
			JP	61047518 A	08/03/1986
			JP	5052889 B	06/08/1993
US	4548079 A	22/10/1985	DE	3246201 A1	14/06/1984
			EP	0111636 A3	05/09/1984
			HU	197446 B	28/03/1989
			JP	6005178 B	19/01/1994
			JP	1884137 C	10/11/1994
			JP	59160720 A	11/09/1984
US	6185996 B1	13/02/2001	NONE		