



- (51) International Patent Classification:
F16H 37/04 (2006.01) F16H 3/54 (2006.01)
- (21) International Application Number:
PCT/SE2018/050067
- (22) International Filing Date:
31 January 2018 (31.01.2018)
- (25) Filing Language: English
- (26) Publication Language: English
- (30) Priority Data:
1750108-1 08 February 2017 (08.02.2017) SE
- (71) Applicant: SCANIA CV AB [SE/SE]; 151 87 Södertälje (SE).
- (72) Inventors: DAHLBÄCK, Märten; Liljevägen 8, 181 61 Lidingö (SE). ARNELÖF, Per; Albatrossvägen 5, 136 66

Vendelsö (SE). FORSBERG, Jörgen; Vildkattens väg 7, 647 35 Mariefred (SE). SLAPAK, Dieter; Förvaltarvägen 20, 151 47 Södertälje (SE).

(74) Agent: HAGSTRÖM, Åsa; Scania CV AB, 151 87 Södertälje (SE).

(81) Designated States (unless otherwise indicated, for every kind of national protection available): AE, AG, AL, AM, AO, AT, AU, AZ, BA, BB, BG, BH, BN, BR, BW, BY, BZ, CA, CH, CL, CN, CO, CR, CU, CZ, DE, DJ, DK, DM, DO, DZ, EC, EE, EG, ES, FI, GB, GD, GE, GH, GM, GT, HN, HR, HU, ID, IL, IN, IR, IS, JO, JP, KE, KG, KH, KN, KP, KR, KW, KZ, LA, LC, LK, LR, LS, LU, LY, MA, MD, ME, MG, MK, MN, MW, MX, MY, MZ, NA, NG, NI, NO, NZ, OM, PA, PE, PG, PH, PL, PT, QA, RO, RS, RU, RW, SA, SC, SD, SE, SG, SK, SL, SM, ST, SV, SY, TH, TJ, TM, TN, TR, TT, TZ, UA, UG, US, UZ, VC, VN, ZA, ZM, ZW.

(54) Title: A GEARBOX FOR VEHICLES

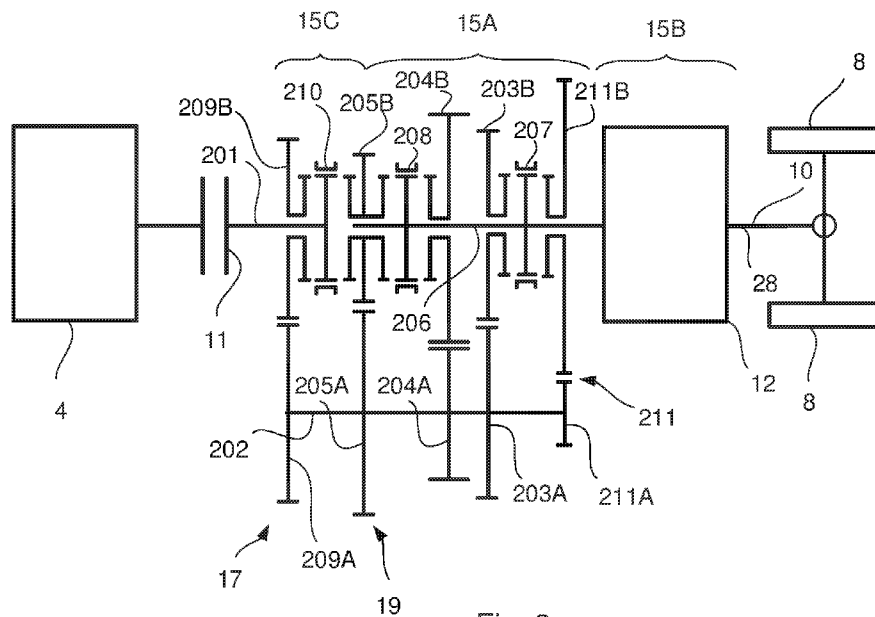


Fig. 2

(57) Abstract: The invention relates to a gearbox for vehicles, comprising a split gearbox (15C), a main gearbox (15A) and a range gearbox (15B), which is shiftable into a low range gear, a high range gear and a reverse gear; the range gearbox (15B) comprising a planetary gear (14) with a ring gear wheel (22), a sun gear wheel (18) and a planet carrier (20), on which at least one planet gear wheel (24) is rotatably mounted, which ring gear wheel (22) and sun gear wheel (18) engage with the at least one planet gear wheel (24); a gearbox housing (12) surrounding the planetary gear (14); a first axially movable coupling sleeve (42) arranged to engage the planet carrier (20) with the gearbox housing (12); and a second axially movable coupling sleeve (43) arranged to engage the ring gear wheel (22) with an output shaft (28) for achieving the reverse gear in the gearbox (2). The split gearbox (15C) and the main gearbox (15A) are together arranged to have a combined largest gear ratio which is larger than the gear ratio in the planetary gear (14) in the range



(84) Designated States (*unless otherwise indicated, for every kind of regional protection available*): ARIPO (BW, GH, GM, KE, LR, LS, MW, MZ, NA, RW, SD, SL, ST, SZ, TZ, UG, ZM, ZW), Eurasian (AM, AZ, BY, KG, KZ, RU, TJ, TM), European (AL, AT, BE, BG, CH, CY, CZ, DE, DK, EE, ES, FI, FR, GB, GR, HR, HU, IE, IS, IT, LT, LU, LV, MC, MK, MT, NL, NO, PL, PT, RO, RS, SE, SI, SK, SM, TR), OAPI (BF, BJ, CF, CG, CI, CM, GA, GN, GQ, GW, KM, ML, MR, NE, SN, TD, TG).

Published:

— *with international search report (Art. 21(3))*

A GEARBOX FOR VEHICLES

BACKGROUND AND PRIOR ART

5

The present invention relates to a gearbox for vehicles and to a vehicle comprising such a gearbox according to the appended claims.

10

In vehicles, and especially for heavier vehicles such as trucks, the gearbox comprises a main gearbox and often also a range gearbox which is connected to the main gearbox. The range gearbox doubles the number of gears in the gearbox. The range gearbox usually includes a planetary gear, which has a low gear and a high gear, so that the gearbox can be divided into a low range gear position and a high range gear position. In the low range gear position a downshift takes place through the planetary gear, and in the high range gear position the gear ratio is 1:1 in the planetary gear.

15

20

The range gearbox is usually provided between the main gearbox and a propeller shaft coupled to the drive wheels of the vehicle. The range gearbox is accommodated in a gearbox housing and comprises an input shaft coupled to the main gearbox, an output shaft and between the input shaft and the output shaft is the planetary gear disposed. The planetary gear usually comprises three components, which are rotatable arranged relative to each other namely a sun gear wheel, a planet carrier with planet gear wheels and a ring gear wheel. With knowledge of the number of teeth of the sun gear wheel and the ring gear wheel the relative speed of the three components can be determined during operation. In a range gearbox the sun gear wheel can be rotatable connected to the input shaft, a number of planet gear wheels which engage said sun gear wheel, which planet gear wheels are rotatable mounted on the planet carrier which is fixedly connected to the output shaft, and an axially displaceable ring gear wheel which surrounds and engages the planet gear wheels.

25

30

In a known range gearbox the low range gear position and high range gear position are obtained by displacing the ring gear wheel axially between the low range gear position, in which the ring gear wheel is rotationally locked relative to the gearbox housing, and high range gear position in which the ring gear wheel is rotatable rela-

tive to the gearbox housing and where the ring gear wheel, the planet gear wheels and the sun gear wheel rotate as a common unity. The known planetary gear comprises two on each side of the ring gear wheel arranged coupling rings and two synchronizing rings arranged on each side of the ring gear wheel. Synchronizer rings is
5 to provide a synchronous shift.

The document WO0155620 discloses a synchronization device in a planetary gear in a range gearbox. The planetary gear includes a sun gear wheel, a planet carrier and a ring gear wheel. The sun gear wheel is rotatable connected with the input shaft and
10 a number of planet gear wheels engaging with the sun gear wheel, which planet gear wheels are rotatable mounted on the planet carrier, which is connected to the output shaft. An axially displaceable ring gear wheel surrounds and meshes with the planet gear wheels. Low and high gear are obtained by the ring gear wheel is displaced axially between low range gear and high range gear.

15

However, there are range gearboxes in which the synchronization devices are replaced with coupling sleeves provided with splines. By controlling the transmission to synchronous speed between the two components to be engaged an axial displacement of the coupling sleeve along the two components is made possible in order to
20 connect them. When the components should be detached the transmission is controlled so that torque balance occurs between the components so that the coupling sleeve is not transmitting torque. It then becomes possible to move the coupling sleeve along the components in order to disengage them from each other.

25 The document US6196944 shows a planetary gear comprising a sun gear wheel, a planet carrier with planet gear wheels and a ring gear wheel. The sun gear wheel may be connected to the input shaft by means of a coupling sleeve in a low range gear position and disengaged from the input shaft in a high range gear position. In the high range gear position the input shaft is connected to of the planet carrier by
30 means of the same coupling sleeve. The ring gear wheel is firmly connected to a gearbox housing. The known planetary gear is arranged in an auxiliary gearbox, having only two gear positions.

The reverse gear in a transmission in a vehicle are often arranged in the main gearbox, which then comprises a gear which is engaged when the vehicle is to be driven in the reversed direction. The gear wheel, which is intended for the reverse gear, causes an elongation of the main gearbox, and an undesired increase in weight of the vehicle.

The document WO2015/183153A1 shows a gearbox for vehicles comprising a range gearbox provided with a first, second and third axially movable coupling sleeves acting on a planetary gear in the range gearbox. Depending on the axial position of the coupling sleeves a reverse gear in the gearbox may be achieved.

The range gearbox must have a diameter and a weight within reasonable limits. If the diameter of the planetary gear in the range gearbox is too large the gearbox will not fit in a vehicle due to geometrical and design limits of the vehicle. An increased diameter of the planetary gear in the range gearbox also results in an increased weight which results in an increased overall weight of the vehicle. This may lead to an increased fuel consumption of the vehicle.

When the planetary gear in the range gearbox is designed with a limited diameter the gear ratio in the range gearbox will be limited to a certain amount. This may affect the driveability of the vehicle in the reverse direction since the vehicle may not be driven with a velocity low enough in the reverse direction when a clutch between an engine and the gearbox is completely engaged.

Since the gearbox must transmit a considerable amount of torque the gear ratio may not be increased by minimizing the diameter of the sun gear wheel. If the sun gear wheel has a very small diameter it may not withstand the considerable amount of torque needed. Also, by using a small sun gear wheel the diameter of the planet gear wheels may get into physical contact with each other due to an increased diameter of the planet gear wheels.

SUMMARY OF THE INVENTION

5 There is a need to further develop a gearbox provided with a reverse gear in a range gearbox. There is also a need to develop a gearbox where all components in the gearbox are utilized effectively so that low energy is required when shifting. There is also a need to develop a gearbox, which exhibits small dimensions relative to the possible transmission of torque. Also, there is a need to develop a gearbox which brings the overall components in the gearbox to a minimum in order to save manufacturing and repair costs.

10 The object of the present invention is therefore to further develop a gearbox provided with a reverse gear in a range gearbox.

15 Another object of the present invention is to provide a gearbox which brings the overall components in the gearbox to a minimum in order to save manufacturing and repair costs.

20 A further object of the invention to provide a gearbox that utilizes all of the transmission components effectively.

A further object of the invention is to provide a gearbox, having small dimensions in relation to possible transmission of torque.

25 A further object of the present invention is to provide a gearbox which requires low energy for shifting.

30 These objects are achieved with the above-mentioned gearbox according to the appended claims.

According to an aspect of the invention a gearbox for vehicles is provided. The gearbox comprises a split gearbox, a main gearbox and a range gearbox, which is shiftable into a low range gear, a high range gear and a reverse gear; the range gearbox comprising a planetary gear with a ring gear wheel, a sun gear wheel and a planet

carrier, on which at least one planet gear is rotatable mounted, which ring gear wheel and sun gear wheel engage with the at least one planet gear; a gearbox housing surrounding the planetary gear; a first axially movable coupling sleeve arranged to engage the planet carrier with the gearbox housing; and a second axially movable coupling sleeve arranged to engage the ring gear wheel with an output shaft for achieving the reverse gear in the gearbox. The split gearbox and the main gearbox are together arranged to have a combined largest gear ratio, which is larger than the gear ratio in the planetary gear in the range gearbox when the planetary gear is shifted into the reverse gear.

10

The vehicle may be driven in the reverse direction when the planetary gear in the range gearbox is shifted into the reverse gear. When the gear ratio in the split gearbox and the main gearbox together have a combined largest gear ratio, which is larger than the gear ratio in the planetary gear in the range gearbox, when the planetary gear is shifted into the reverse gear, the driveability of the vehicle in the reverse direction may be acceptable even though a clutch between an engine and the gearbox is completely engaged. With such a combined or common gear ratio in the split gearbox and the main gearbox in relation to the gear ratio in the range gearbox, the vehicle may be driven in the reverse direction in a velocity which is low enough to have control over the vehicle. With such a combined gear ratio the range gearbox may have a limited diameter and still the driveability of the vehicle in the reverse direction is not affected. A limited diameter of the range gearbox will bring the overall components in the gearbox to a minimum, and thus manufacturing and repair costs will be saved. The sun gear wheel in the planetary gear may be provided with a diameter which withstands the considerable amount of torque needed. Thus, there is no need to increase the gear ratio of the planetary gear in the range gearbox by minimizing the diameter of the sun gear wheel.

15

20

25

30

The gearbox is also designed so that the combined or common gear ratio in the split gearbox and the main gearbox have a ratio spread that is large enough to drive the vehicle in the reverse direction in a velocity which is low enough to have control over the vehicle. The definition of the ratio spread is the largest common gear ratio of the split gearbox and the main gearbox divided with the minimum common gear ratio of the split gearbox and the main gearbox.

According to a further aspect of the invention the split gearbox and the main gearbox are together arranged to have a combined largest gear ratio over 4:1 and the planetary gear in the range gearbox is arranged to have a gear ratio less than 4:1 when shifted into the reverse gear. When the combined largest gear ratio in the split gearbox and the main gearbox together is over 4:1 the driveability of the vehicle in the reverse direction is acceptable even though the gear ratio less than 4:1 in the planetary gear in the range gearbox. With such a common gear ratio in the split gearbox and the main gearbox in relation to the gear ratio in the range gearbox, the vehicle may be driven in the reverse direction in a velocity which is low enough to have control over the vehicle.

According to a further aspect of the invention the gearbox is arranged to have an overlapping gear ratio in at least the highest gear in the low range gear position and at least the lowest gear in a high range gear position. The overlapping gear ratio will increase the total gear ratio in the gearbox. Such a gearbox brings the overall components in the gearbox to a minimum and utilizes all of the transmission components effectively.

According to a further aspect of the invention the gearbox is arranged to have an overlapping gear ratio in the two highest gear in the low range gear position and the two lowest gear in a high range gear position. The overlapping gear ratio in the two highest gears in the low range gear position and the two lowest gears in a high range gear position will increase the total gear ratio in the gearbox. This way, the vehicle may be driven in the reverse direction in a velocity which is low enough to have control over the vehicle. The driveability of the vehicle in the reverse direction will be very good.

When the gearbox is arranged with a split gearbox a main gearbox and a range gearbox when the planetary gear is shifted into the reverse gear, the gear wheels in the split gearbox and the main gearbox may be so designed that an overlapping gear ratio is achieved. The overlapping gear ratio in the two highest gears in the low range gear position and the two lowest gears in a high range gear position will increase the

total gear ratio spread in the gearbox. Such gearbox brings the overall components in the gearbox to a minimum and utilizes all of the transmission components effectively.

5 According to a further aspect of the invention the split gearbox comprises an incoming gear and a split gear; and the gear ratio of the split gear is larger than the gear ratio of the incoming gear. When the gear ratio of the split gear is larger than the gear ratio of the incoming gear the overall gear ratio of the gearbox may be increased.

10 According to a further aspect of the invention the gear ratio of the incoming gear is in the range 0,9:1 - 1,1:1. When the gear ratio of the incoming gear is in the range 0,9:1 - 1,1:1, the overall gear ratio of the gearbox may be increased and the vehicle may be driven in the reverse direction in a velocity which is low enough to have control over the vehicle. Thus, the driveability of the vehicle in the reverse direction will be acceptable. The overall dimensions of the gearbox can be reduced. In addition, there
15 is no need to increase the gear ratio of the planetary gear in the range gearbox by minimizing the diameter of the sun gear wheel.

20 According to a further aspect of the invention the gear ratio of the incoming gear is in the range 0,95:1 - 1,05:1. When the gear ratio of the incoming gear is in the range 0,95:1 - 1,05:1, the overall gear ratio of the gearbox may be increased and the vehicle may be driven in the reverse direction in a velocity which is low enough to have control over the vehicle. Thus, the driveability of the vehicle in the reverse direction will be improved. The overall dimensions of the gearbox can be reduced further. In addition, there is no need to increase the gear ratio of the planetary gear in the range
25 gearbox by minimizing the diameter of the sun gear wheel.

30 According to a further aspect of the invention the gear ratio of the incoming gear is 1:1. When the gear ratio of the incoming gear is 1:1, the overall gear ratio of the gearbox may be increased and the vehicle may be driven in the reverse direction in a velocity which is low enough to have control over the vehicle. Thus, the driveability of the vehicle in the reverse direction will be very good. The overall dimensions of the gearbox can be reduced even further. In addition, there is no need to increase the gear ratio of the planetary gear in the range gearbox by minimizing the diameter of the sun gear wheel.

According to a further aspect of the invention the split gearbox and the main gearbox are together arranged to have a combined ratio spread larger than 5 and the planetary gear is arranged to have a gear ratio less than 4:1 when shifted into the reverse gear. With such a combined ratio spread in the split gearbox and the main gearbox together with such a gear ratio in the range gearbox, the vehicle may be driven in the reverse direction in a velocity which is low enough to have control over the vehicle. Thus, the driveability of the vehicle in the reverse direction is acceptable. The overall dimensions of the gearbox can be reduced. In addition, there is no need to increase the gear ratio of the planetary gear in the range gearbox by minimizing the diameter of the sun gear wheel.

According to a further aspect of the invention the split gearbox and the main gearbox are together arranged to have a combined ratio spread in the range 5,5 - 6,3 and the planetary gear is arranged to have a gear ratio in the range 3,1:1 - 3,9:1 when shifted into the reverse gear. With such a combined ratio spread in the split gearbox and the main gearbox together with such a gear ratio in the range gearbox, the vehicle may be driven in the reverse direction in a velocity which is low enough to have control over the vehicle. Thus, the driveability of the vehicle in the reverse direction is acceptable. The overall dimensions of the gearbox can be reduced. In addition, there is no need to increase the gear ratio of the planetary gear in the range gearbox by minimizing the diameter of the sun gear wheel.

According to a further aspect of the invention the split gearbox and the main gearbox are together arranged to have a combined ratio spread in the range 5,7 - 6,1 and the planetary gear is arranged to have a gear ratio in the range 3,3:1 - 3,6:1 when shifted into the reverse gear. With such a combined ratio spread in the split gearbox and the main gearbox together with such a gear ratio in the range gearbox, the vehicle may be driven in the reverse direction in a velocity which is low enough to have control over the vehicle. Thus, the driveability of the vehicle in the reverse direction will be improved. The overall dimensions of the gearbox can be reduced further. In addition, there is no need to increase the gear ratio of the planetary gear in the range gearbox by minimizing the diameter of the sun gear wheel.

The above-mentioned objects are also achieved by a vehicle, including the above-mentioned gearbox, according to the appended claims.

BRIEF DESCRIPTION OF THE DRAWINGS

5

Below is a description of, as examples, preferred embodiments of the invention with reference to the enclosed drawings, in which:

10 Fig. 1 shows schematically a vehicle in a side view with a gearbox according to the invention,

Fig. 2 shows schematically a cross section of the main gearbox and a split gearbox according to the invention,

15 Fig. 3 shows schematically a cross section of the gearbox according to the invention in a low range gear position,

Fig. 4 shows schematically a cross section of the gearbox according to the invention in a high range gear position, and

20

Fig. 5 shows schematically a cross section of the gearbox according to the invention in a reverse position.

DETAILED DESCRIPTION OF PREFERRED EMBODIMENTS OF THE INVENTION

25

Fig. 1 shows schematically a vehicle 1 in a side view, provided with a powertrain 3, which comprises a gearbox 2 according to the invention. The powertrain 3 also comprises an internal combustion engine 4, a propeller shaft 10 and drive wheels 8. The drive wheels 8 are coupled to the gearbox 2 via the propeller shaft 10. The gearbox 2
30 comprises a range gearbox 15B which aims to double the number of gear opportunities. The gearbox 2 is surrounded by a gearbox housing 12.

Fig. 2 shows a schematic sectional view of a gearbox 2 according to the present invention. The gearbox 2 comprises a main gearbox 15A, a split gearbox 15C and the

range gearbox 15B. The range gearbox 15B is shiftable into a low range gear, a high range gear and a reverse gear, which will be explained in more detail below. The vehicle 1 may be driven in the reverse direction when the range gearbox 15B is shifted into the reverse gear. The combined gear ratio in the split gearbox 15C and the main gearbox 15A together may be larger than the gear ratio in a planetary gear 14 (fig. 3) in the range gearbox 15B, when the planetary gear 14 is shifted into the reverse gear. The split gearbox 15C and the main gearbox 15A may together be arranged to have a combined largest gear ratio over 4:1 and the planetary gear 14 in the range gearbox 15B may be arranged to have a gear ratio less than 4:1 when shifted into the reverse gear. With such a common or combined gear ratio in the split gearbox 15C and the main gearbox 15A in relation to the gear ratio in the range gearbox 15B, the vehicle 1 may be driven in the reverse direction in a velocity which is low enough to have control over the vehicle 1 even though a clutch 11 between the engine 4 and the gearbox 2 is completely engaged. The velocity of the vehicle 1 driven in the reverse direction at a gear with the gear ratio mentioned above and with the clutch 11 completely engaged will be below three km/h.

In order to arrange the above mentioned gear ratio in the gearbox 2, the two highest gears in the low range gear position and the two lowest gears in a high range gear position may be overlapping. Thus, the two highest gears in the low range gear positions should have substantially the same gear ratio as the two lowest gears in a high range gear position. However, at least the highest gear in the low range gear position and at least the lowest gear in a high range gear position should be overlapping. Thus, at least the highest gear in the low range gear positions should have substantially the same gear ratio as at least the lowest gear in a high range gear position. This overlapping gear ratio will increase the ratio spread in the gearbox 2. This will bring the overall number of components in the gearbox 2 to a minimum and also utilize all of the transmission components effectively. The definition of the ratio spread is the largest common or combined gear ratio of the split gearbox 15C and the main gearbox 15A divided with the minimum common or combined gear ratio of the split gearbox 15C and the main gearbox 15A.

The split gearbox 15C comprises an incoming gear 17 and a split gear 19. The gear ratio of the split gear 19 may be larger than the gear ratio of the incoming gear 17, so

that the overall gear ratio in the gearbox 2 may be increased. As an example, the gear ratio of the incoming gear 17 may be in the range 0,9:1 - 1,1:1 and thus the gear ratio of the split gear 19 may be larger than 0,9:1 - 1,1:1. When the gear ratio of the incoming gear 17 is in the range 0,9:1 - 1,1:1, the overall gear ratio of the gearbox 2 may be increased and the vehicle 1 may be driven in the reverse direction in a low enough velocity for a driver of the vehicle 1 to have control over the vehicle 1. Thus, the driveability of the vehicle 1 in the reverse direction will be acceptable. The overall dimensions of the gearbox 2 can be reduced. Also, there is no need to increase the gear ratio of the planetary gear 14 in the range gearbox 15B by minimizing the diameter of the sun gear wheel 18.

According to another example the gear ratio of the incoming gear 17 may be in the range 0,95:1 - 1,05:1 and thus the gear ratio of the split gear 19 may be larger than 0,95:1 - 1,05:1. When the gear ratio of the incoming gear 17 is in the range 0,95:1 - 1,05:1, the overall gear ratio of the gearbox 2 may be increased and the vehicle 1 may be driven in the reverse direction in a low enough velocity for a driver of the vehicle 1 to have control over the vehicle 1. Thus, the driveability of the vehicle 1 in the reverse direction will be improved. The overall dimensions of the gearbox 2 can be reduced further. Also, there is no need to increase the gear ratio of the planetary gear 14 in the range gearbox 15B by minimizing the diameter of the sun gear wheel 18.

The gear ratio of the incoming gear 17 may also be substantially 1:1 and thus the gear ratio of the split gear 19 may be larger than 1:1. When the gear ratio of the incoming gear 17 is substantially 1:1, the overall gear ratio of the gearbox 2 may be increased and the vehicle 1 may be driven in the reverse direction in a velocity which is low enough for a driver of the vehicle 1 to have control over the vehicle 1. Thus, the driveability of the vehicle 1 in the reverse direction will be very good. The overall dimensions of the gearbox 2 can be reduced even further. Also, there is no need to increase the gear ratio of the planetary gear 14 in the range gearbox 15B by minimizing the diameter of the sun gear wheel 18.

The split gearbox 15C and the main gearbox 15A may together be arranged to have a combined ratio spread in the range 5,5 - 6,3 and the planetary gear 14 may be ar-

ranged to have a gear ratio in the range 3,1:1 - 3,9:1 when shifted into the reverse gear. With such a combined ratio spread in the split gearbox 15C and the main gearbox 15A together with such a gear ratio in the range gearbox 15B, the vehicle 1 may be driven in the reverse direction in a velocity which is low enough for a driver of the vehicle 1 to have control over the vehicle 1. Thus, the driveability of the vehicle 1 in the reverse direction will be acceptable. The overall dimensions of the gearbox 2 can be reduced. In addition, there is no need to increase the gear ratio of the planetary gear 14 in the range gearbox 15B by minimizing the diameter of the sun gear wheel 18.

10

Alternatively, the split gearbox 15C and the main gearbox 15A may together be arranged to have a combined ratio spread in the range 5,7 - 6,1 and the planetary gear 14 is arranged to have a gear ratio in the range 3,3:1 - 3,6:1 when shifted into the reverse gear. With such a combined ratio spread in the split gearbox 15C and the main gearbox 15A together with such a gear ratio in the range gearbox 15B, the vehicle 1 may be driven in the reverse direction in a low enough velocity for a driver of the vehicle 1 to have control over the vehicle 1. Thus, the driveability of the vehicle 1 in the reverse direction will be improved. The overall dimensions of the gearbox 2 can be reduced further. Also, there is no need to increase the gear ratio of the planetary gear 14 in the range gearbox 15B by minimizing the diameter of the sun gear wheel 18.

15

20

According to fig. 2 the main gearbox 15A alone can be set to four different gear ratios. The range gearbox 15B is arranged downstream the main gearbox 15A. The range gearbox 15B is surrounded by a gearbox housing 12 and is discussed further with reference to figures 3 - 5. The split gearbox 15C is located upstream the main gearbox 15A in the direction of torque from combustion engine 4 to drive wheels 8. The split gearbox 15C provides for each gear of the main gearbox 15A into two gear steps with different gear ratios in order to provide more gear ratios of the gearbox 2.

25

30

Instead of a disengageable clutch 11 an arrangement with first and second electrical machines (not disclosed) may be arranged to rotate and brake a planetary gear (not disclosed) arranged in the powertrain 3 and located upstream the gearbox 2. In such an arrangement the first electrical machine may be arranged at a sun gear wheel (not

disclosed) of the planetary gear 14, and the second electrical machine should be arranged at the first ring gear wheel (not disclosed) of the planetary gear. The first and second electrical machines may be the power source or may form part of the power source.

5

With regard to the main gearbox 15A, a layshaft 202 comprises gear wheels 203A, 204A, 205A that are rotatably fixed to the layshaft 202. For example, gear wheel 203A represents the second gear, gear wheel 204A the first gear and gear wheel 205A the third gear. A mainshaft 206 comprises corresponding gear wheels 203B, 204B, 205B which rotate freely in relation to the mainshaft 206, but which can be selectively locked for rotation with the mainshaft 206 in order to engage a gear. For example, the second main gearbox gear can be engaged by manoeuvring a first main sleeve 207, arranged to rotate with the mainshaft 206, to a position where the gear wheel 203B is engaged, i.e. to the left in the figure, thereby bringing the gear wheel 203B to rotate the mainshaft 206 and thereby also engaging the layshaft 202 to the mainshaft 206 via gear wheel 203A. Each pair of gear wheels on the layshaft 202 and mainshaft 206 representing a gear ratio.

The first main gearbox gear can be engaged by disengaging the first main sleeve 207 from gear wheel 203B and instead moving a second main sleeve 208 to a position to the right in the figure where, instead, gear wheel 204B is engaged, thereby bringing the gear wheel 204B to rotate the mainshaft 206. Correspondingly, the third main gearbox gear can be engaged by manoeuvring the second main sleeve 208 to a position to the left in the figure where, instead gear wheel 205B is engaged, thereby setting the main gearbox 15A to third gear. Each of the first through third gears is used for a plurality of the total number of gears provided by the gearbox 2 as a whole. For example, the first gear of the main gearbox 15A will be used for first and second gear of the gearbox 2, low and high split, low range, and also for seventh and eighth gear, low and high split, and high range. The different gears in the split gearbox 15C and the main gearbox 15A may also be used when the range gearbox 15B is shifted into the reverse gear.

The main gearbox 15A is also provided with a crawler gear 211 comprising a crawler gear wheel pair 211A and 211B. The crawler gear wheel 211A is arranged on the

layshaft and the crawler gear wheel 211B may rotate freely in relation to the mainshaft 206, but can be selectively locked for rotation with the mainshaft 206 in order to engage the crawler gear 211.

- 5 Further, with regard to the split gearbox 15C, the incoming gear 17 comprises a incoming gear wheel 209A that, similar to the above, is rotatably fixed to the layshaft 202 and a corresponding incoming gear wheel 209B rotating freely in relation to the input shaft 201 but which can be selectively locked for rotation with the input shaft 201 through a split sleeve 210, which may be provided with a split synchronising unit.
- 10 The split sleeve 210 can further be used to connect the gearbox input shaft 201 to gear wheel 205B directly. The incoming gear 17 comprising the gear wheel pair 209A-B can together with the split sleeve 210 thereby provide two different split gear ratios for each gear of the main gearbox 15A.
- 15 When e.g. the first gear is engaged, the split sleeve 210 is arranged to engage split gear wheel 205B. This will have the result that the input shaft 201 is directly connected to the split gear wheel 205B, which via the split gear wheel 205A establishes a first gear ratio between input shaft 201 and layshaft 202. Gear wheel 205B, however, is not connected to the mainshaft 206, but the layshaft 202 is connected to mainshaft
- 20 206 through gear wheel pair 204A-B and by means of the second main sleeve 208.

When the second gear is engaged, i.e. high split of first main gearbox gear, the vehicle 1 is, instead, driven with gear wheel pair 209A-B engaged, resulting in a second gear ratio between an input shaft 201 and layshaft 202. The gear wheel 204B is still

25 engaged by the second main sleeve 208 according to the above, thereby extending the range of each gear.

This split can be performed for each gear of the main gearbox 15A. However, when the coupling unit 210 engages gear wheel 205B and also the second main sleeve

30 208 engages gear wheel 205B a gear ratio of 1:1 through the split gearbox 15C and the main gearbox 15A is obtained.

The crawler gear 211 is according to the invention engaged when a reverse gear is engaged in the range gearbox 15B. A first reverse gear is engaged when also the

split sleeve 210 is arranged to engage split gear wheel 205B. This will have the result that the input shaft 201 is directly connected to the split gear wheel 205B, which via the split gear wheel 205A establishes a gear ratio between input shaft 201 and lay-shaft 202.

5

The crawler gear 211 is also engaged when the gearbox 2 is shifted into the forward direction when the range gearbox 15B is shifted into the low range position, but not in the high range position. In comparison to other gears used when driving a vehicle 1 in the forward direction, such as the gear wheel pair 204A-B representing the first gear in the main gearbox 15A, the reverse gear is less used and also the crawler gear 211 in the forward direction is less used. Also, the torque transferred through the crawler gear 211 is less than the torque transferred through other gears in the gearbox 2, such as the gear wheel pair 204A-B, due to the high gear ratio in the crawler gear 211. For this reason wear of the crawler gear wheel pair 211A and 211B will be limited.

10
15

According to fig. 2 the crawler gear 211 is arranged in the main gearbox 15A. However, it may also be possible to arrange the crawler gear 211 in the split gearbox 15C, so that the crawler gear wheel 211A is arranged on the input shaft 201 and the crawler gear wheel 211B is arranged on the layshaft 202.

20

Fig. 3 shows a schematic sectional view of a gearbox 2 of the present invention. The gearbox 2 comprises a main gearbox 15A, a split gearbox 15C and the range gearbox 15B. The range gearbox 15B comprises a planetary gear 14 which has a low and a high gear, so that the switching capability of the gearbox 2 can be divided into a low range gear position and a high range gear position. In a first gear position corresponding to the low range gear position a downshift takes place in the planetary gear 14. In the high range gear position the gear ratio is 1:1 in the planetary gear 14. The planetary gear 14 is in Fig. 3 shifted into the first gear position, corresponding to the low range gear position.

25
30

The range gearbox 15B is accommodated in the gearbox housing 12 and comprises a range gearbox input shaft 16 which may be a mainshaft 26 of the main gearbox 15A. The planetary gear 14 comprises three main components which are rotatable

arranged in relation to each other, namely a sun gear wheel 18, a planet carrier 20 and a ring gear wheel 22. A number of planet gear wheels 24 are rotatable arranged with bearings on the planet carrier 20. With knowledge of the number of teeth 32 of sun gear wheel 18 and the ring gear wheel 22, the relative gear ratio of the three
5 components can be determined. The sun gear wheel 18 is rotatable connected to the input shaft 16 and the planet gear wheels 24 engage the sun gear wheel 18. The ring gear wheel 22 surrounds and engages the planet gear wheels 24.

10 A first axially displaceable coupling sleeve 42 is in a first gear position arranged to connect the transmission case 12 with the ring gear wheel 22 and in a second gear position arranged to disconnect the transmission case 12 from the ring gear wheel 22. The first axially movable coupling sleeve 42 is in the first gear position arranged to disconnect the input shaft 16 from the planet carrier 20.

15 A second axially displaceable coupling sleeve 43 is in a third gear position arranged to couple the ring gear wheel 22 with an output shaft 28 of the gearbox 2. The output shaft 28 is coupled to the propeller shaft 10 of the vehicle 1. In the third gear position, corresponding to a reverse gear, the first axially displaceable coupling sleeve 42 is arranged to disconnect the input shaft 16 from the planet carrier 20 and instead is
20 arranged to interconnect the planet carrier 20 with the gearbox housing 12. In the first and second gear positions the second axially displaceable coupling sleeve 43 is arranged to interconnect the planet carrier 20 with the output shaft 28.

25 The first axially displaceable coupling sleeve 42 is on an inner surface provided with first splines 50 arranged to interact with the corresponding first splines 50 arranged on the ring gear wheel 22 and on the periphery of a projection 52 which is fixedly connected to the transmission housing 12. The first splines 50 on the first axially displaceable coupling sleeve 42 are also arranged to cooperate with corresponding first splines 50 arranged on the input shaft 16. Corresponding first splines 50 disposed on
30 the input shaft 16 are made on the periphery of a first sprocket 46 which is mounted on a shaft 38 for the sun gear wheel 18. The first splines 50 on the first axially displaceable coupling sleeve 42 are also arranged to cooperate with corresponding first splines 50 arranged on the planet carrier 20. Corresponding first splines 50 disposed

on the planet carrier 20 are made on the periphery of a second sprocket 44 which is mounted on the planet carrier 20.

5 The second axially displaceable coupling sleeve 43 is on an inner surface provided with second splines 51 which are arranged to cooperate with corresponding second splines 51 arranged on the ring gear wheel 22, the planet carrier 20 and the output shaft 28. The corresponding second splines 51 arranged on the planet carrier 20 are formed on the periphery of a third sprocket 49 which is mounted on the planet carrier 20. The corresponding second splines 51 provided on the output shaft 28 are formed
10 on the periphery of a fourth sprocket 53 which is mounted on the output shaft 28.

The low gear in the gearbox 2 is obtained by displacing the first coupling sleeve 42, so that the ring gear wheel 22 is connected to the gearbox housing 12. The axial displacement of the first and second coupling sleeves 42, 43 are provided with a first
15 and second shift fork 60, 61 arranged in an outside circumferential groove 62 in the respective coupling sleeve 42, 43. The first shift fork 60 is influenced by a first power means 66 and the second shift fork 61 is influenced by a second power means 67. The first and second power means 66, 67 may be a pneumatic or hydraulic cylinder. The shift forks 60, 61 and power means 66, 67 are schematically shown in Fig. 3.
20

Preferably, the coupling sleeves 42, 43 each has a low weight, which means that there is a need of low energy and force to move the respective coupling sleeves 42, 43 when shifting gears. This allows a quick gear shifting between the different gear positions in the gearbox 2.
25

Fig. 4 shows a schematic sectional view of the range gearbox 15B in the second gear position or high range gear position in which the first coupling sleeve 42 is shifted to the right in Fig. 4 for connecting the input shaft 16 to the planet carrier 20. In this position the first coupling sleeve 42 has disconnected the ring gear wheel 22 of
30 the gearbox housing 12. The transmission of torque from the input shaft 16 to the output shaft 28 is in the high range gear position via the input shaft 16 and planet carrier 20 and further to the output shaft 28 via the second coupling sleeve 43, so that the gear ratio through the planetary gear 14 becomes 1:1. As an alternative, the

second axially movable coupling sleeve 43 may in the second gear position be arranged to engage the ring gear wheel 22 with the output shaft 28.

In fig. 5 the range gearbox 15B has been shifted into the third gear position, which is the reverse gear. The second axially displaceable sleeve 43 is in the third gear position arranged to connect the ring gear wheel 22 with output shaft 28. Thus, the second coupling sleeve 43 is shifted by the second shift fork 61, so that the ring gear wheel 22 is connected to the output shaft 28. The first coupling sleeve 42 is shifted by the first shift fork 60 to couple the planet carrier 20 with the gearbox housing 12. The planet carrier 20 is provided with a second sprocket 44 arranged for connecting the planet carrier 20 with the gearbox housing 12 by means of the first coupling sleeve 42. The displacement of the respective clutch sleeve 42, 43 is performed when the input and the output shaft 16, 28 are stationary, which corresponds to a stationary operating state of the vehicle 1, when the range gearbox 15B is included in the transmission 3 of a vehicle 1. In order to provide a stationary position of the input shaft 16 the clutch 11 of the vehicle 1 is transferred to a disconnected mode. When the range gearbox 15B is operated in the third gear position, torque is transmitted from the input shaft 16 to the sun gear wheel 18 and further to the planet gear wheels 24 which transmits the torque to the ring gear wheel 22 and further to the output shaft 28 via the second coupling sleeve 43. The planet carrier 20 is stationary as the first coupling sleeve 42 connects the planet carrier 20 with the gearbox housing 12.

With the gear ratio mentioned above, the range gearbox 15B may have a limited diameter and still the driveability of the vehicle 1 in the reverse direction is not adversely affected. A limited diameter of the range gearbox 15B will bring the overall components in the gearbox 2 to a minimum, and thus manufacturing and repair costs will be saved. The sun gear wheel 18 in the planetary gear 14 may be provided with a diameter which withstands the considerable amount of torque needed. Thus, there is no need to increase the gear ratio of the planetary gear 14 in the range gearbox 15B by minimizing the diameter of the sun gear wheel 18.

An electronic control unit 70 is coupled to the powertrain 3 to achieve the gear shifting above. A number of not shown speed sensors in the powertrain 3 may be connected to the control unit 70. Another computer 72 may also be connected to the

control unit 70. The control unit 70 may be a computer with appropriate software for this purpose. The control unit 70 and/or the computer 72 comprise a computer program P, which can include routines to control the gearbox 2 of the present invention. The program P may be stored in an executable form or compressed form in a
5 memory M and/or in a read/write memory. Preferably there is provided a computer program product comprising a program code stored on a, by a computer readable medium for performing the gear shifting above, when said program is run on the control unit 70 or another computer 72 connected to the control unit 70. Said code may be non-volatile, stored in said computer readable medium.

10

The components and features specified above may within the framework of the invention be combined between the different embodiments specified.

Claims

1. A gearbox for vehicles, comprising a split gearbox (15C), a main gearbox (15A)
5 and a range gearbox (15B), which is shiftable into a low range gear, a high range
gear and a reverse gear; the range gearbox (15B) comprising a planetary gear (14)
with a ring gear wheel (22), a sun gear wheel (18) and a planet carrier (20), on which
at least one planet gear wheel (24) is rotatable mounted, which ring gear wheel (22)
and sun gear wheel (18) engage with the at least one planet gear wheel (24); a gear-
10 box housing (12) surrounding the planetary gear (14); a first axially movable coupling
sleeve (42) arranged to engage the planet carrier (20) with the gearbox housing (12);
and a second axially movable coupling sleeve (43) arranged to engage the ring gear
wheel (22) with an output shaft (28) for achieving the reverse gear in the gearbox (2),
characterized in that the split gearbox (15C) and the main gearbox (15A) are to-
15 gether arranged to have a combined largest gear ratio, which is larger than the gear
ratio in the planetary gear (14) in the range gearbox (15B) when the planetary gear
(14) is shifted into the reverse gear.
2. The gearbox according to claim 1, **characterized in that** the split gearbox (15C)
20 and the main gearbox (15A) are together arranged to have a combined largest gear
ratio over 4:1 and the planetary gear (14) in the range gearbox (15B) is arranged to
have a gear ratio less than 4:1 when shifted into the reverse gear.
3. The gearbox according to any of the preceding claims, **characterized in that** the
25 gearbox (2) is arranged to have an overlapping gear ratio in at least the highest gear
in the low range gear position and at least the lowest gear in a high range gear posi-
tion.
4. The gearbox according to claim 3, **characterized in that** the gearbox (2) is ar-
30 ranged to have an overlapping gear ratio in the two highest gears in the low range
gear position and the two lowest gears in a high range gear position.
5. The gearbox according to any of the preceding claims, **characterized in that** the
split gearbox (15C) comprises an incoming gear (17) and a split gear (19); and in that

the gear ratio of the split gear (19) is larger than the gear ratio of the incoming gear (17).

5 6. The gearbox according to claim 5, **characterized in that** the gear ratio of the incoming gear (17) is in the range 0,9:1 - 1,1:1.

7. The gearbox according to claim 5, **characterized in that** the gear ratio of the incoming gear (17) is in the range 0,95:1 - 1,05:1.

10 8. The gearbox according to claim 5, **characterized in that** the gear ratio of the incoming gear (17) is 1:1.

15 9. The gearbox according to any of the preceding claims, **characterized in that** the split gearbox (15C) and the main gearbox (15A) are together arranged to have a combined ratio spread larger than 5 and the planetary gear (14) is arranged to have a gear ratio less than 4:1 when shifted into the reverse gear.

20 10. The gearbox according to any of the preceding claims, **characterized in that** the split gearbox (15C) and the main gearbox (15A) are together arranged to have a combined ratio spread in the range 5,5 - 6,3 and the planetary gear (14) is arranged to have a gear ratio in the range 3,1:1 - 3,9:1 when shifted into the reverse gear.

25 11. The gearbox according to any of the claims 1-9, **characterized in that** the split gearbox (15C) and the main gearbox (15A) are together arranged to have a combined ratio spread in the range 5,7 - 6,1 and the planetary gear (14) is arranged to have a gear ratio in the range 3,3:1 - 3,6:1 when shifted into the reverse gear.

12. A vehicle, **characterized in that** the vehicle (1) is provided with a gearbox (2) according to any of the preceding claims.

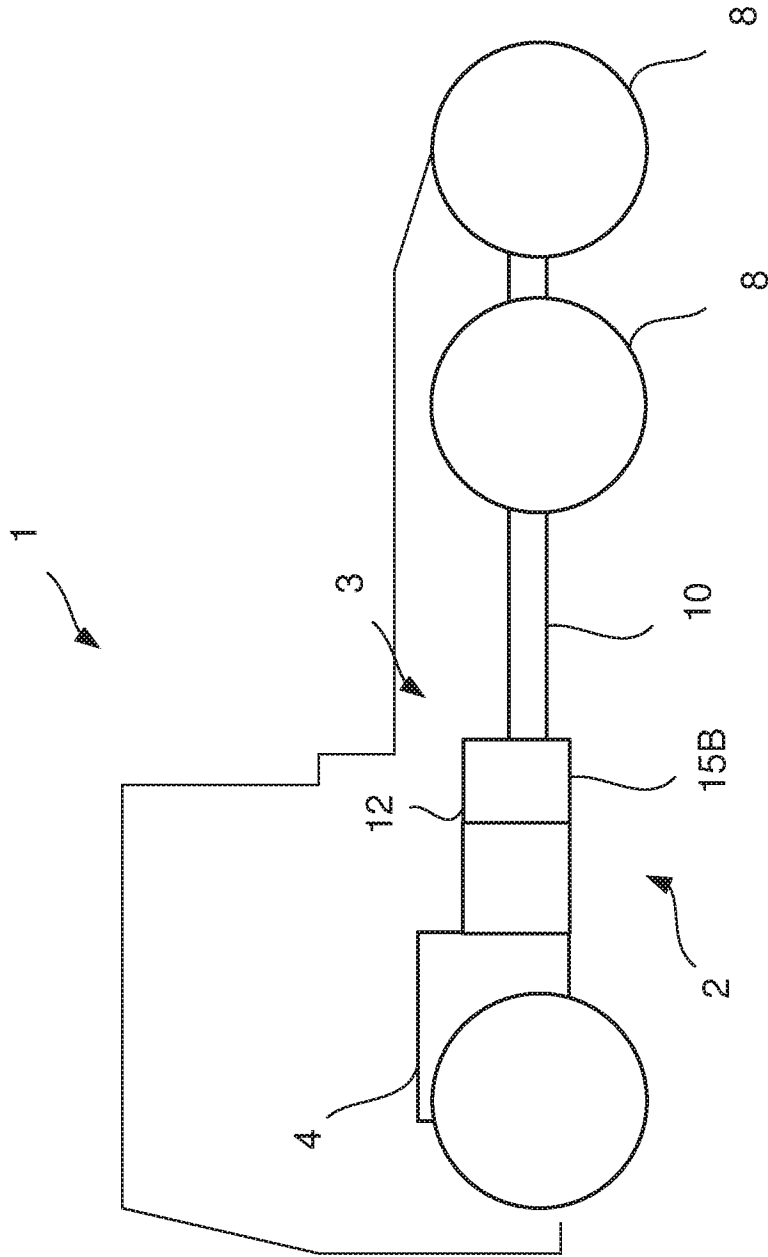


Fig. 1

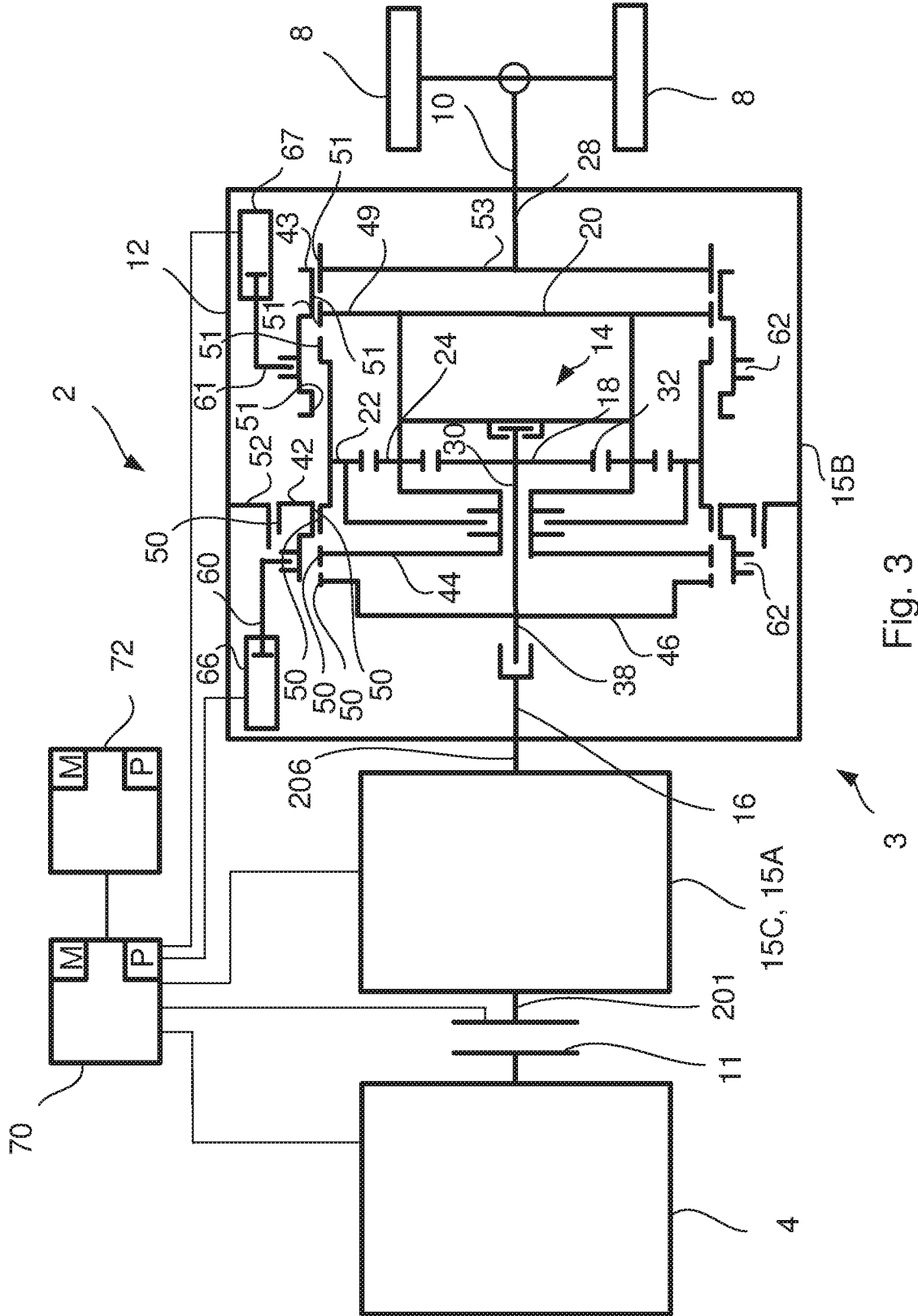


Fig. 3

4/5

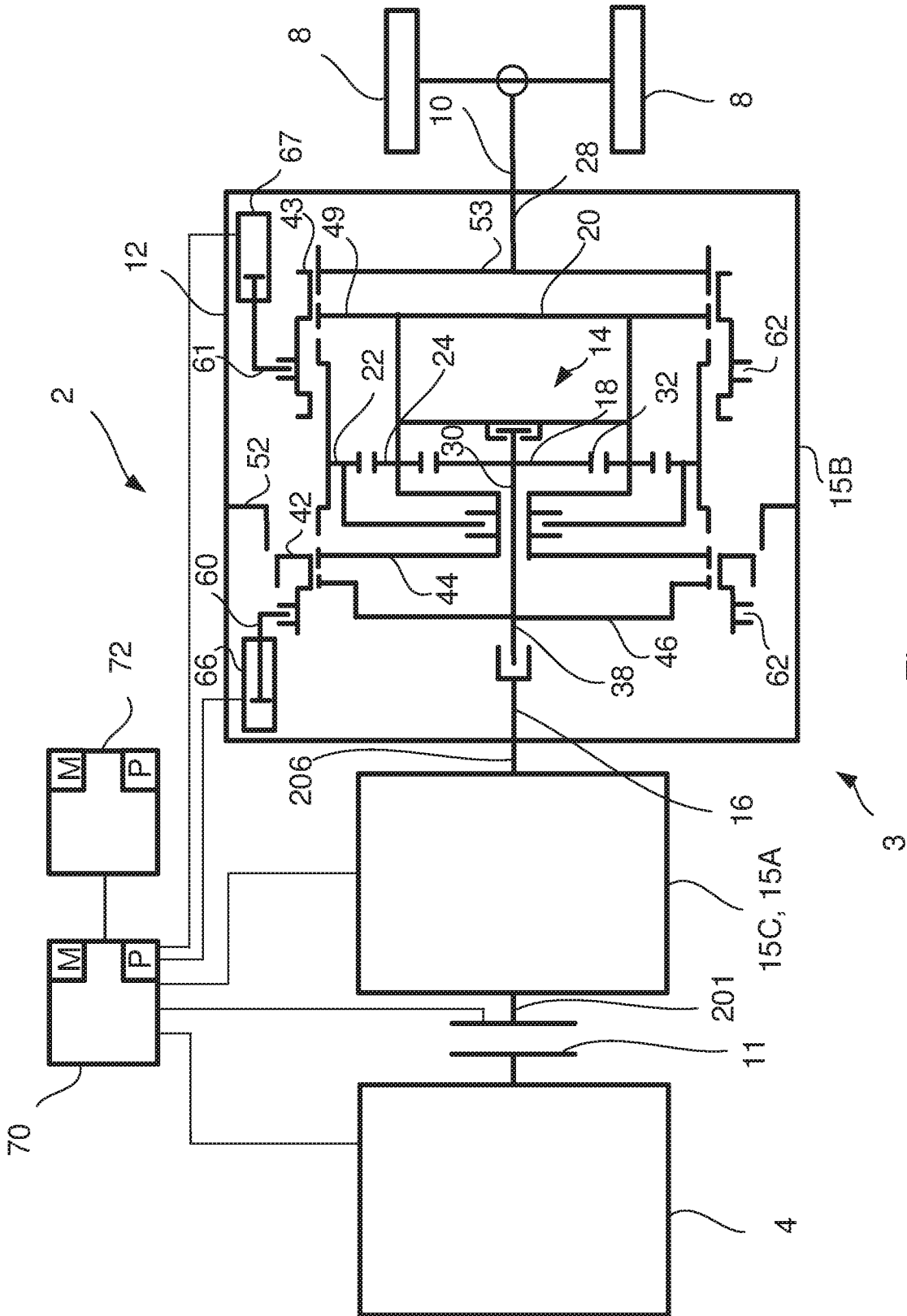
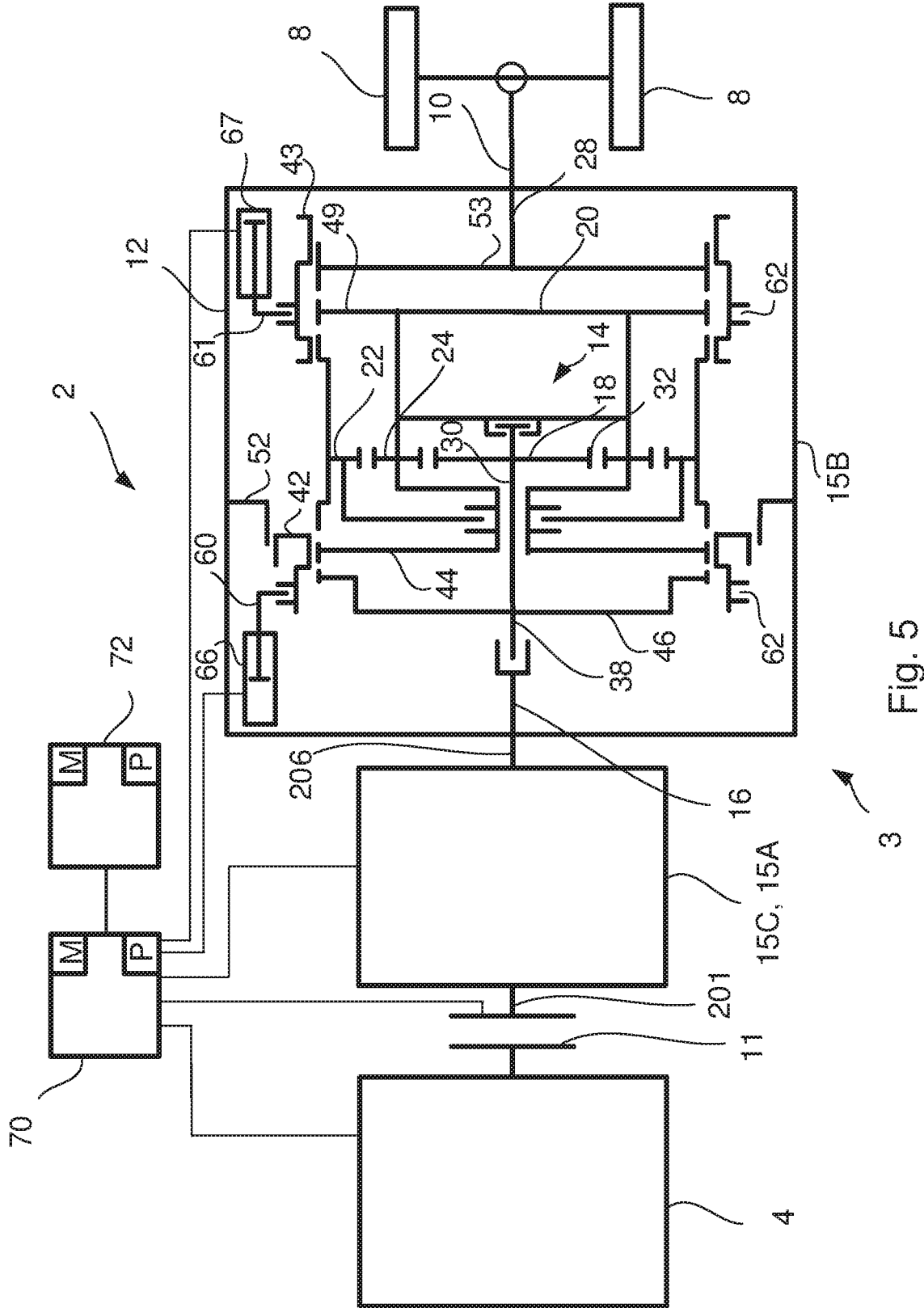


Fig. 4



3 Fig. 5

INTERNATIONAL SEARCH REPORT

International application No.
PCT/SE2018/050067

A. CLASSIFICATION OF SUBJECT MATTER		
IPC: see extra sheet		
According to International Patent Classification (IPC) or to both national classification and IPC		
B. FIELDS SEARCHED		
Minimum documentation searched (classification system followed by classification symbols)		
IPC: F16H		
Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched		
SE, DK, FI, NO classes as above		
Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)		
EPO-Internal, PAJ, WPI data		
C. DOCUMENTS CONSIDERED TO BE RELEVANT		
Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	WO 2015183159 A1 (SCANIA CV AB), 3 December 2015 (2015-12-03); abstract; page 10, line 18 - page 11, line 7; page 11, line 27 - page 14, line 24; figures 2-6 --	1-12
A	EP 0601645 A1 (FORD NEW HOLLAND LTD), 15 June 1994 (1994-06-15); abstract; page 1, line 17 - line 39 --	3, 4
A	JP 2008014403 A (MITSUBISHI AGRICULT MACH), 24 January 2008 (2008-01-24); figures 5-6; PAJ abstract --	3, 4
<input checked="" type="checkbox"/> Further documents are listed in the continuation of Box C. <input checked="" type="checkbox"/> See patent family annex.		
* Special categories of cited documents:		
"A" document defining the general state of the art which is not considered to be of particular relevance	"T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention	
"E" earlier application or patent but published on or after the international filing date	"X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone	
"L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)	"Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art	
"O" document referring to an oral disclosure, use, exhibition or other means	"&" document member of the same patent family	
"P" document published prior to the international filing date but later than the priority date claimed		
Date of the actual completion of the international search	Date of mailing of the international search report	
29-03-2018	04-04-2018	
Name and mailing address of the ISA/SE Patent- och registreringsverket Box 5055 S-102 42 STOCKHOLM Facsimile No. + 46 8 666 02 86	Authorized officer Johan Åhman Telephone No. + 46 8 782 28 00	

INTERNATIONAL SEARCH REPORT

International application No.
PCT/SE2018/050067

C (Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT		
Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	DE 102013204172 A1 (ZAHNRADFABRIK FRIEDRICHSHAFEN), 18 September 2014 (2014-09-18); abstract; paragraphs [0131]-[0133]; figures 26-27 --	1-12
A	EP 2982884 A1 (ZAHNRADFABRIK FRIEDRICHSHAFEN), 10 February 2016 (2016-02-10); abstract; paragraphs [0015]-[0018], [0038]-[0039]; figures 1-3 --	1-12
A	US 20140364269 A1 (KALTENBACH JOHANNES), 11 December 2014 (2014-12-11); abstract; paragraphs [0098]-[0109]; figures 3a,3b,3c --	1-12
A	DE 102013204918 A1 (ZAHNRADFABRIK FRIEDRICHSHAFEN), 25 September 2014 (2014-09-25); abstract; paragraphs [0138]-[0142]; figures 26-28 --	1-12
A	DE 102016007249 A1 (SCANIA CV AB), 12 January 2017 (2017-01-12); abstract; paragraphs [0035]-[0037]; figure 2 --	1-12
A	WO 2015183153 A1 (SCANIA CV AB), 3 December 2015 (2015-12-03); page 16, line 13 - page 17, line 8; figures 2-5; cited in the application --	1-12
A	US 20150219193 A1 (NITSCH MATTHIAS ET AL), 6 August 2015 (2015-08-06); abstract; paragraph [0019]; figure 1 --	1-12
A	US 20040040397 A1 (RAZZACKI SYED T), 4 March 2004 (2004-03-04); abstract; claim 1 -- -----	1-12

Continuation of: second sheet

International Patent Classification (IPC)

F16H 37/04 (2006.01)

F16H 3/54 (2006.01)

INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No.

PCT/SE2018/050067

WO	2015183159 A1	03/12/2015	KR	20170009924 A	25/01/2017
			SE	1550665 A1	28/11/2015
			US	20170089437 A1	30/03/2017
EP	0601645 A1	15/06/1994	DE	69308271 D1	03/04/1997
			US	5337624 A	16/08/1994
JP	2008014403 A	24/01/2008	NONE		
DE	102013204172 A1	18/09/2014	EP	2971854 A1	20/01/2016
			US	20160025189 A1	28/01/2016
			WO	2014139743 A1	18/09/2014
EP	2982884 A1	10/02/2016	DE	102014215551 A1	11/02/2016
			US	9759299 B2	12/09/2017
			US	20160040768 A1	11/02/2016
US	20140364269 A1	11/12/2014	DE	102013210429 A1	08/01/2015
			EP	2899427 A1	29/07/2015
			US	9267577 B2	23/02/2016
DE	102013204918 A1	25/09/2014	EP	2976550 A1	27/01/2016
			US	20160061304 A1	03/03/2016
			WO	2014146840 A1	25/09/2014
DE	102016007249 A1	12/01/2017	BR	102016013118 A2	21/02/2017
			SE	1550992 A1	09/01/2017
			SE	539177 C2	02/05/2017
WO	2015183153 A1	03/12/2015	EP	3149360 A4	21/03/2018
			SE	538915 C2	14/02/2017
			SE	1450626 A1	28/11/2015
			US	9869370 B2	16/01/2018
			US	20170089429 A1	30/03/2017
US	20150219193 A1	06/08/2015	CN	204784504 U	18/11/2015
			DE	102014202080 A1	06/08/2015
			SE	1550078 A1	06/08/2015
			US	9423013 B2	23/08/2016
US	20040040397 A1	04/03/2004	US	20050021209 A1	27/01/2005
			US	6889131 B2	03/05/2005
			US	6827666 B2	07/12/2004