



(51) International Patent Classification:
F16L 37/14 (2006.01)

(21) International Application Number:
PCT/EP2009/064704

(22) International Filing Date:
5 November 2009 (05.11.2009)

(25) Filing Language: English

(26) Publication Language: English

(30) Priority Data:
08168530.7 6 November 2008 (06.11.2008) EP

(71) Applicant (for all designated States except US): **NYBERG, Bo Erik** [SE/CH]; Schneitstrasse 71, CH-6315 Oberägeri (CH).

(72) Inventor; and

(75) Inventor/Applicant (for US only): **BOHMAN, Peter** [SE/SE]; P.O. Box 245, S-541 25 Skövde (SE).

(74) Agent: **ALBIHNS AB**; P.O. Box 5581, Valhallavägen 117, S-114 85 Stockholm (SE).

(81) Designated States (unless otherwise indicated, for every kind of national protection available): AE, AG, AL, AM, AO, AT, AU, AZ, BA, BB, BG, BH, BR, BW, BY, BZ, CA, CH, CL, CN, CO, CR, CU, CZ, DE, DK, DM, DO, DZ, EC, EE, EG, ES, FI, GB, GD, GE, GH, GM, GT, HN, HR, HU, ID, IL, IN, IS, JP, KE, KG, KM, KN, KP, KR, KZ, LA, LC, LK, LR, LS, LT, LU, LY, MA, MD, ME, MG, MK, MN, MW, MX, MY, MZ, NA, NG, NI, NO, NZ, OM, PE, PG, PH, PL, PT, RO, RS, RU, SC, SD, SE, SG, SK, SL, SM, ST, SV, SY, TJ, TM, TN, TR, TT, TZ, UA, UG, US, UZ, VC, VN, ZA, ZM, ZW.

(84) Designated States (unless otherwise indicated, for every kind of regional protection available): ARIPO (BW, GH, GM, KE, LS, MW, MZ, NA, SD, SL, SZ, TZ, UG, ZM, ZW), Eurasian (AM, AZ, BY, KG, KZ, MD, RU, TJ, TM), European (AT, BE, BG, CH, CY, CZ, DE, DK, EE, ES, FI, FR, GB, GR, HR, HU, IE, IS, IT, LT, LU, LV, MC, MK, MT, NL, NO, PL, PT, RO, SE, SI, SK, SM, TR), OAPI (BF, BJ, CF, CG, CI, CM, GA, GN, GQ, GW, ML, MR, NE, SN, TD, TG).

Published:

— with international search report (Art. 21(3))

(54) Title: COUPLING

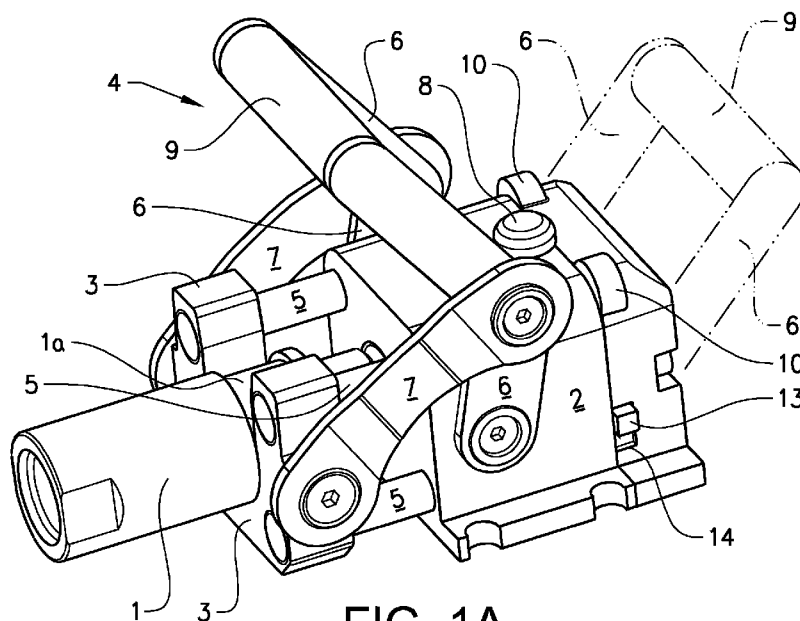


FIG. 1A

(57) Abstract: A high pressure fluid coupling assembly comprises a nipple (1) with a neck 1a to be held in a yoke 3, which is pulled towards a socket box 2 by a lever mechanism 4. When the nipple is pulled into complete engagement with the socket, the final downward movement of the lever 6 pushes a locking plate 10 into locking engagement with the distal ends of guide bolts 5, evenly distributed about the nipple and extending from the yoke and sliding through the socket box 2 to the locking plate 10, which is in contact with the side of the socket box 2 facing away from the yoke. The locking plate 10, guide bolts 5, yoke 3 and socket box 2 take up the nipple/socket separating force generated by the very high hydraulic pressure instead of the arms and joints of the lever mechanism 4.

COUPLING

TECHNICAL FIELD

5 BACKGROUND ART

High pressure fluid couplings when closed usually are subjected to extreme forces on the closing mechanism and its arms during use. Demolition machines, hydraulic hammers and mining machines often use large diameter hoses ($\geq 1'' = 26 \text{ mm}$) and routinely develop pressures of up to 350 bar
10 during use. This puts enormous forces on the arms and joints of the closing mechanism resulting either in failure in the joints or bending of the arms or extremely bulky dimensioning of the arms and joints.

SUMMARY OF THE INVENTION

15 The present invention relates to a high pressure fluid coupling assembly of the type described in the preamble to the accompanying Claim 1.

The present invention intends to avoid the problems of excessive force on the arms and joints of the closing mechanism of the coupling, when the nipple and socket are fully engaged and the system is fully pressurized
20 creating a force tending to separate the nipple from the socket. This is achieved by a high pressure fluid coupling assembly of the type described by way of introduction which has the characterizing features given in Claim 1. An assembly of this type provides a secure locking of the nipple in the socket. According to a preferred embodiment the load, when the coupling is
25 under pressure, is transferred from the closing mechanism to the locking plate and the distributed guide bolts. According to other preferred embodiments of the invention, the locking plate is engaged with the narrower neck portions of the guide bolts in a particularly simple, efficient and effective manner, simplifying engagement and disengagement and providing optimal
30 taking up of the forces tending to separate the nipple from the socket when the system is under high fluid pressure. The preferred embodiments allow the closing mechanism to be dimensioned in a less bulky and more functional

manner since the closing mechanism and its joints will not be subjected to excessive forces when the system is fully pressurized.

BRIEF DESCRIPTION OF THE DRAWINGS

- 5 The invention will now be described with reference to an illustrative example shown in the accompanying figures of which:
- Fig. 1A shows the assembly of the invention with the nipple resting in the yoke in a state prior to coupling to the socket,
- Fig. 1 B shows the assembly of the invention with the nipple and the socket
10 in a fully coupled state,
- Figs. 2A and 2B show longitudinally sectioned partial views of the assemblies in decoupled and coupled states respectively, corresponding to Figs. 1A and 1B respectively,
- Figs. 3A and 3B show cross sectional views in the decoupled and coupled
15 states respectively as viewed in the direction of the arrows A in Figs. 1A and 1B respectively,
- Figs. 4A and 4B show decoupled and coupled views corresponding to views Figs. 1A and 1B respectively but in the reverse direction and not showing the socket box or the lever mechanism,
- 20 Figs. 5A and 5B show views corresponding to Figs. 4A and 4B respectively but showing the lever mechanism, the socket box and the nipple in longitudinal section.

25 DETAILED DESCRIPTION

The invention will be described with reference here to a single non-limiting illustrative example, where the same reference numerals are used for the same parts in the various views.

- 30 Fig. 1A shows a fluid coupling according to the invention where a typical fluid coupling nipple 1 on a hose (not shown) has been placed with its neck 1a of smaller diameter in a yoke 3 having a slot of minimally larger inner diameter

than the diameter of the neck 1a. A socket box 2 connects to a further hydraulic line (not shown in Figs. 1-2) . The yoke 3 has four guide bolts 5 symmetrically distributed about the central axis of the assembly. The four guide bolts 5 extend through the box 2. The nipple 1 held in the yoke 3 is moved to its fully coupled state by pulling the lever 6 of the mechanism 4 to the right as shown in the figures to the position shown in Fig. 1B, with the yoke 3 retracted against the box 2. The connecting arm 7 between the lever 6 and the yoke 3 will be in a horizontal position in the fully coupled state.

10 As best can be seen in Figs. 2A-B and 4A-B the coupling is provided with four guide bolts 5 (only two shown in Figs. 2A-B) which are solidly fixed in the yoke 3 and extend in bores through the socket box 2. These guide bolts 5 are moved to the right through the socket box as seen in Figs. 2A-B and to the left as seen in Figs. 4A-B when the lever mechanism is actuated to pull the yoke 3 closer to the socket box 2.

A locking plate 10 is provided on the side of the socket box 2 opposite to the yoke and it is provided with four openings 11,12 each having a wider portion 11a, 12a and a narrower portion 11b, 12b. The lower openings 12 in this embodiment happen to be open downwards, this being an optional design choice. The locking plate is displaceable between an upper position shown in Figures 1A, 2A, 3A, 4A and 5A and a lower position shown in Figures 1B, 2B, 3B, 4B and 5B. When in the upper position, the distal ends of the guide bolts 5 pass through the wider portions 11a, 12a of the openings in the guide plate as the yoke 3 is brought closer to the socket box 2.

The locking plate 10 is provided with two lateral projections 13, both of which are visible in Figs. 3A and 3B. The lateral projections 13 extend through holes 14 in the sides of the socket box 2. When the lever 6 and the arm 7 have pulled the distal ends of the guide bolts through the wider portions of the openings 11, 12 in the locking plate, the locking phase of the lever actuation takes place when the lever 6 has rotated enough to begin a

downward movement and come into contact with the lateral projections 13. This final downward movement of the lever will push the locking plate downwards as indicated inter alia by the arrow B in Fig. 2A. Each of the guide bolts is provided near its distal end with a neck portion 5a of narrower diameter which corresponds to the width of the narrower portion 11b, 12b of each opening 11, 12 in the locking plate 10. As the locking plate is pushed downwards it locks each guide bolt 5 axially by fitting over the narrow neck 5a of each guide bolt.

As can best be seen in Figs. 3A and 3B this downward movement of the locking plate 10 is made against the force of two compression springs 15. When the locking plate has been pushed down to its final downward position (Figs. 1B, 2B, 3B, 4B and 5B), a lock pin 16, which is biased to the right as shown in Figs. 5A and 5B by a compression spring (not shown), moves into the notch 17 on the top of the locking plate, thereby holding the plate in its locking position. The lock pin 16 is connected to a knob 8.

When in this locking position, the locking plate 10 which rests flush against the wall of the socket box 2, the guide bolts 5 and the yoke 3 take up the extreme force attempting to separate the nipple 1 from the socket box 2 due to the high hydraulic fluid pressure in the hoses. In this case the force is evenly distributed by the locking around the central axis of the socket box, normal to this axis, and is not absorbed, as it otherwise would be, by the joints and arms of the lever mechanism 4, giving rise to shear stresses and eventual failure.

For decoupling, the locking plate is released by pushing the knob 8 towards the yoke 3, thereby moving the lock pin 16 out of engagement with the locking plate 10, allowing the lock plate 10 to move upwards by the force of the springs 15 if the lever 6 is raised. The lever 6 is then moved to its original position as shown in Figs. 1A, 2A, 3A, 4A and 5A and the nipple can be removed from the yoke.

The nipple 1 is provided with a spring biased central plug 18, which prevents escape of pressurized fluid when the nipple is not coupled into the socket. As it is coupled into the socket of the socket box 2, a central projection 19 on a hose connected portion 20 pushes against the plug 18, thereby pushing the valve element 18 from its seat. A compression spring 21 connects plug 18 to a second valve element 22 which is also immediately unseated if the back pressure in the supply hose to the nipple 1 is not excessive. Should the supply hose connected to the nipple 1 have excessive residual pressure therein, perhaps due to a black hose lying in the sun, the movement of the plug 18 eventually actuates a tiny central venting valve (not visible here) which vents out the overpressure in the hose, enabling full opening the fluid connection between the nipple and the socket when the nipple is fully engaged.

15

It will be understood that other configurations of the components are possible within the scope of the claims. For instance a locking plate could be arranged to be flush against the yoke instead, where the guide bolts would be fixed solidly in the socket box and extend through bores in the yoke and have narrower neck portions at their ends penetrating the yoke. This would preserve the same general inventive principle of the locking plate and distribution of the load. It is also conceivable that the locking plate could move in a rotary direction, instead of downwards, being actuated by a single side projection, with the openings for the pins being oriented in rotary alignment instead and the lock plate being spring biased for rotational movement instead of vertical movement.

25

It is conceivable as well that two hoses stacked on top of each other can be coupled using the same yoke, made twice as high as in the illustrated embodiment. This single yoke, with a single lever mechanism and a single socket box, but with two sockets, and a single locking plate can achieve secure coupling of two hoses with a single operation prior to turning on the

30

high pressure in the supply hose. The single locking plate, lying flush against the socket box, will then take up the extreme force in both hoses tending to decouple them from their sockets. The two hoses can also conceivably be arranged side-by-side in a double yoke, all according to the principle of the invention. Any practical multiple number of hoses is also conceivable.

CLAIMS

1. High pressure fluid coupling assembly comprising:

- a hose nipple (1) with a distal flange (1b) and a neck (1a) of narrower diameter than the distal flange,
- 5 • a box (2) having a socket into which the hose nipple (1) fits upon coupling,
- a yoke (3), which is open to one side, is of narrower inner diameter than said distal flange (1b) and can receive and hold said nipple neck (1a),
- 10 • a lever mechanism (4) actuatable to pull said yoke (3) axially from an open extended position toward said box (2) to a closed retracted position, to pull said hose nipple (1), when its neck (1a) is held in said yoke, into full engagement with said socket,

characterized by:

- 15 • a plurality of axially extending guide bolts (5), distributed about a central axis of the coupling and fixed to the one of the yoke (3) or the box (2) and extending through the other of the box or the yoke;
- a locking plate (10) movable perpendicular to said central axis
- 20 between an unlocked position permitting axial movement of said guide bolts (5) and a locking position locking each of the guide bolts from axial movement, and thereby locking said nipple (1) in full engagement with said socket.

- 25 2. Assembly according to Claim 1, characterized in that said guide bolts (5) are solidly fixed to said yoke (3) and extend through said socket box (2).

- 30 3. Assembly according to Claim 2, characterized in that the locking plate (10) is disposed on the side of the box (2) facing away from said yoke (3).

4. Assembly according to one of the preceding claims characterized in that said locking plate (10) is disposed to be moved to its locking position when the lever mechanism (4) has pulled the yoke (3) to its closed retracted position.

5

5. Assembly according to Claim 4, characterized in that the lever mechanism (4) moves said locking plate (10) from its open position to its locking position subsequent to pulling the yoke (3) to its closed retracted position and the nipple (1) to full engagement with the socket.

10

6. Assembly according to one of the preceding claims characterized in that fluid pressure acting to separate the nipple (1) from the socket is absorbed by the guide bolts (4) and the locking plate (10) when in its locking position and not by the lever mechanism (4).

15

7. Assembly according to Claim 6, characterized in that a spring biased lock pin (15) holds said locking plate (10) in its locking position.

20

8. Assembly according to Claim 7, characterized in that said lock pin (15) can be displaced manually against its spring bias to release the locking plate (10) from its locking position.

25

9. Assembly according to one of the preceding claims, characterized in that the said guide bolts (5) proximate their distal ends (5b) are each provided with a narrower neck portion (5a) engaging the locking plate when it is moved to its locking position.

30

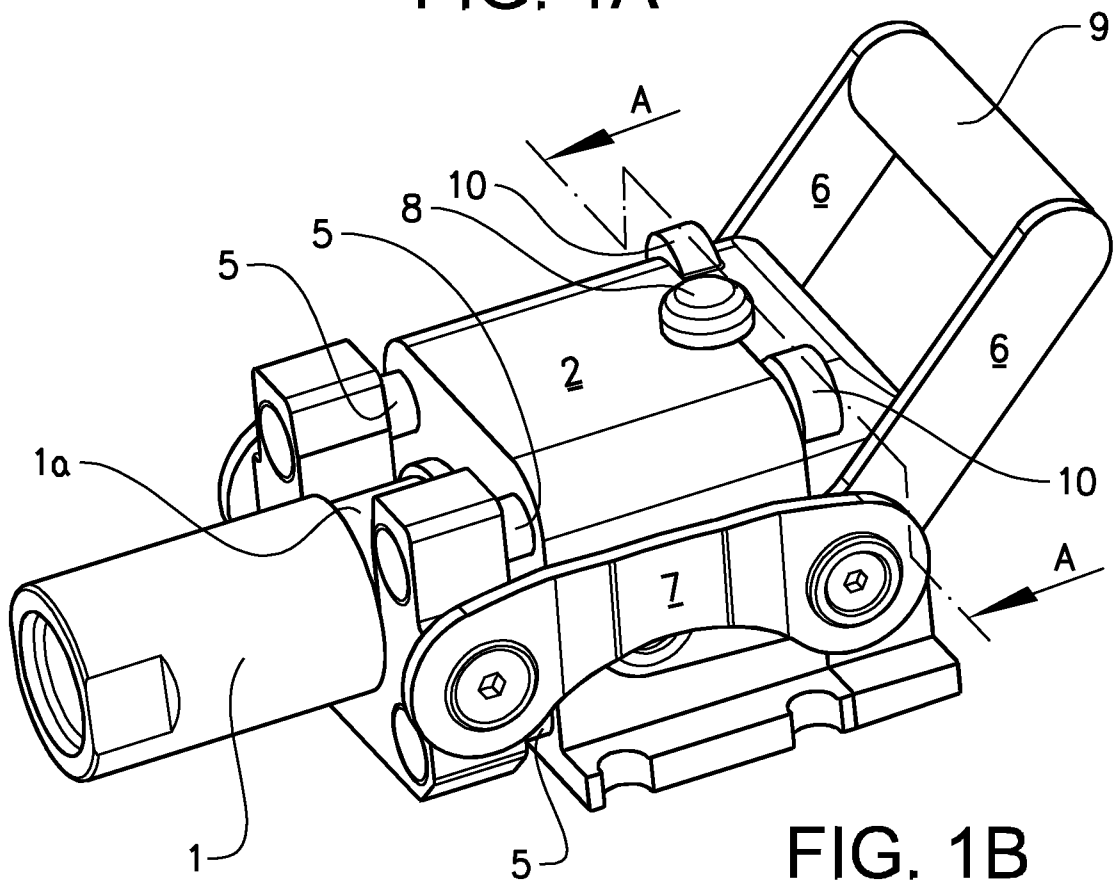
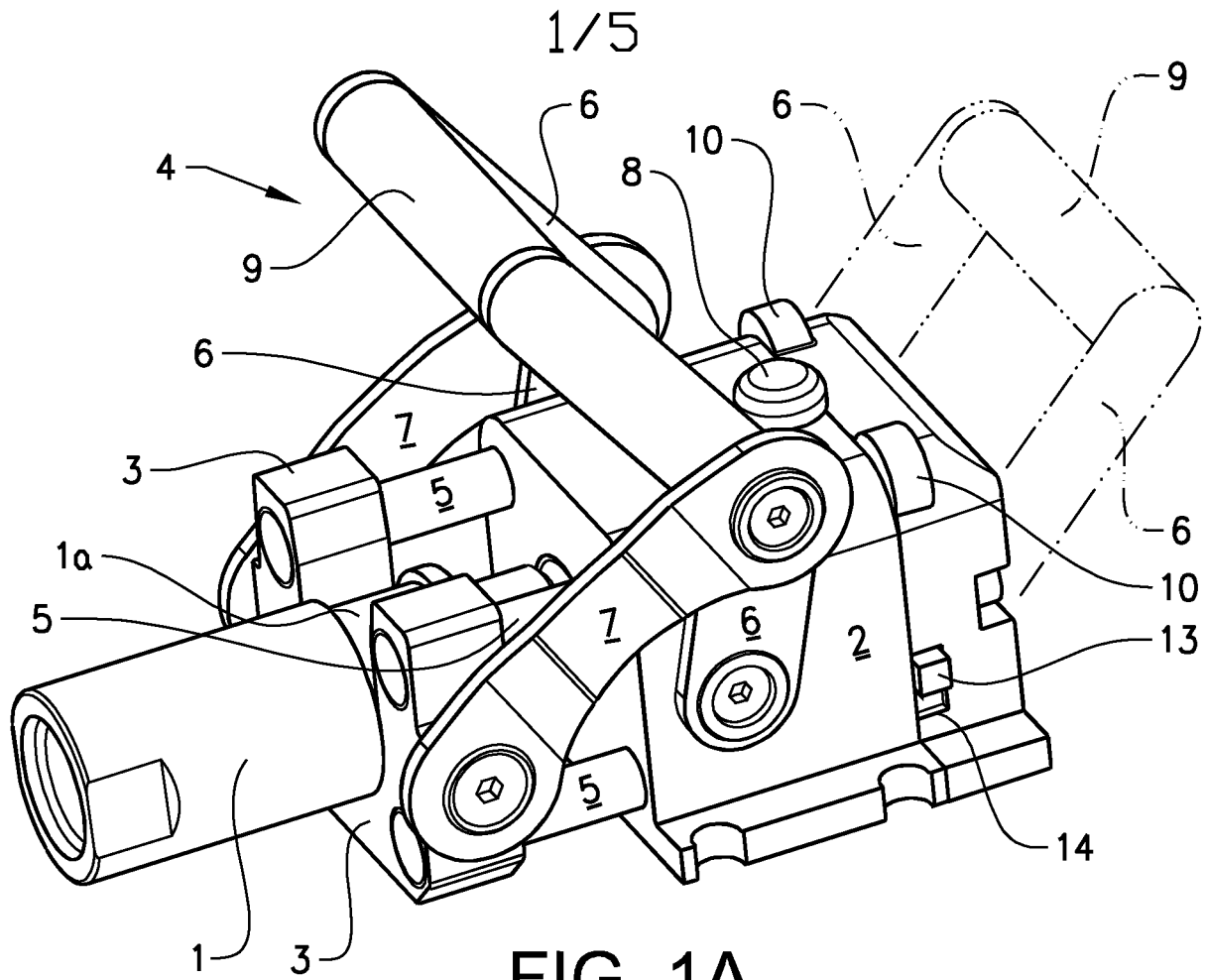
10. Assembly according to one Claim 9, characterized in that the locking plate (10) is provided with an opening for each of the guide bolts (5), each opening having a wider portion allowing insertion of the distal end (5b) of each guide bolt therethrough when the locking plate (10) is in its unlocked

position and a narrower slot portion grasping the neck portion (5a) of each guide bolt (5) when the locking plate (10) is in its locking position.

5 11. Nipple engaging unit comprising a yoke (3), a nipple socket box (2), a lever mechanism (4), guide bolts (5) and a locking plate (10) as disclosed in one of Claims 1-10.

10 12. Nipple engaging mechanism according to Claim 11, characterized in that said yoke and nipple socket box are adapted to engage two nipples at the same time.

15



2/5

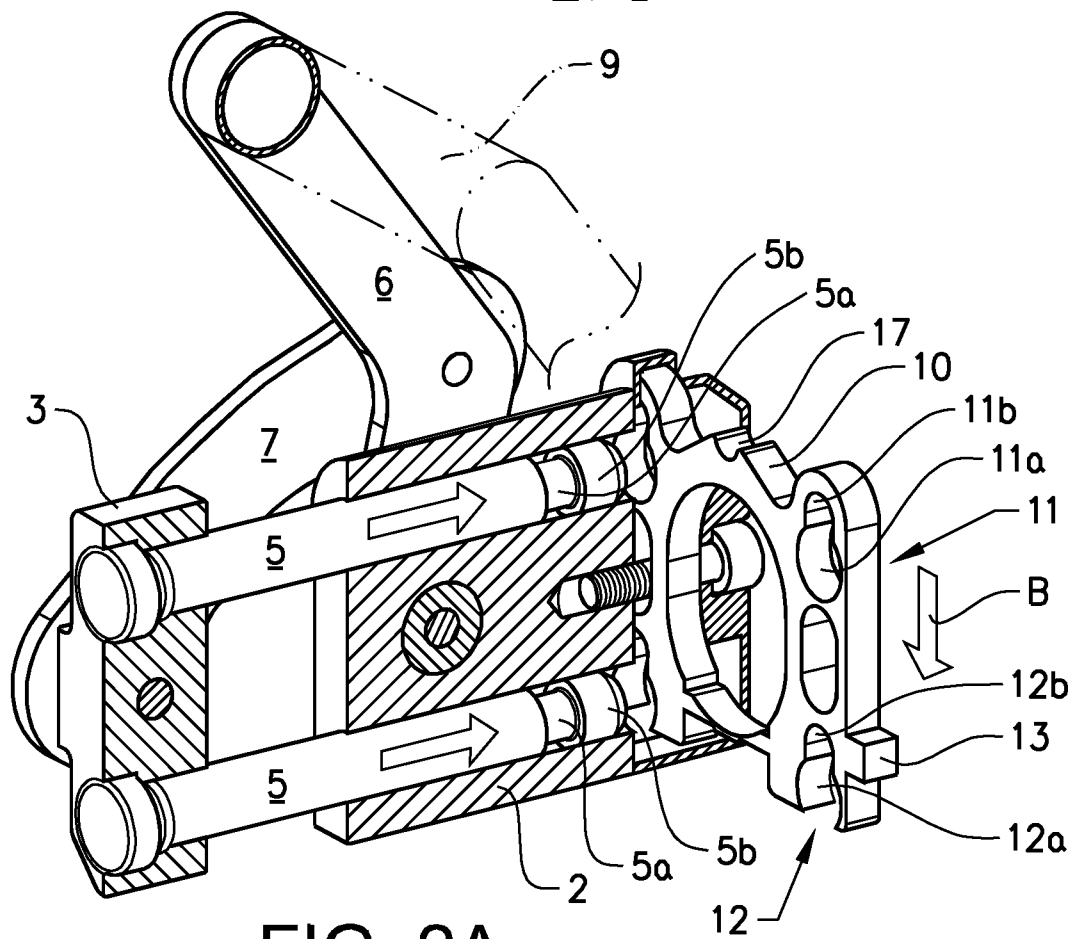


FIG. 2A

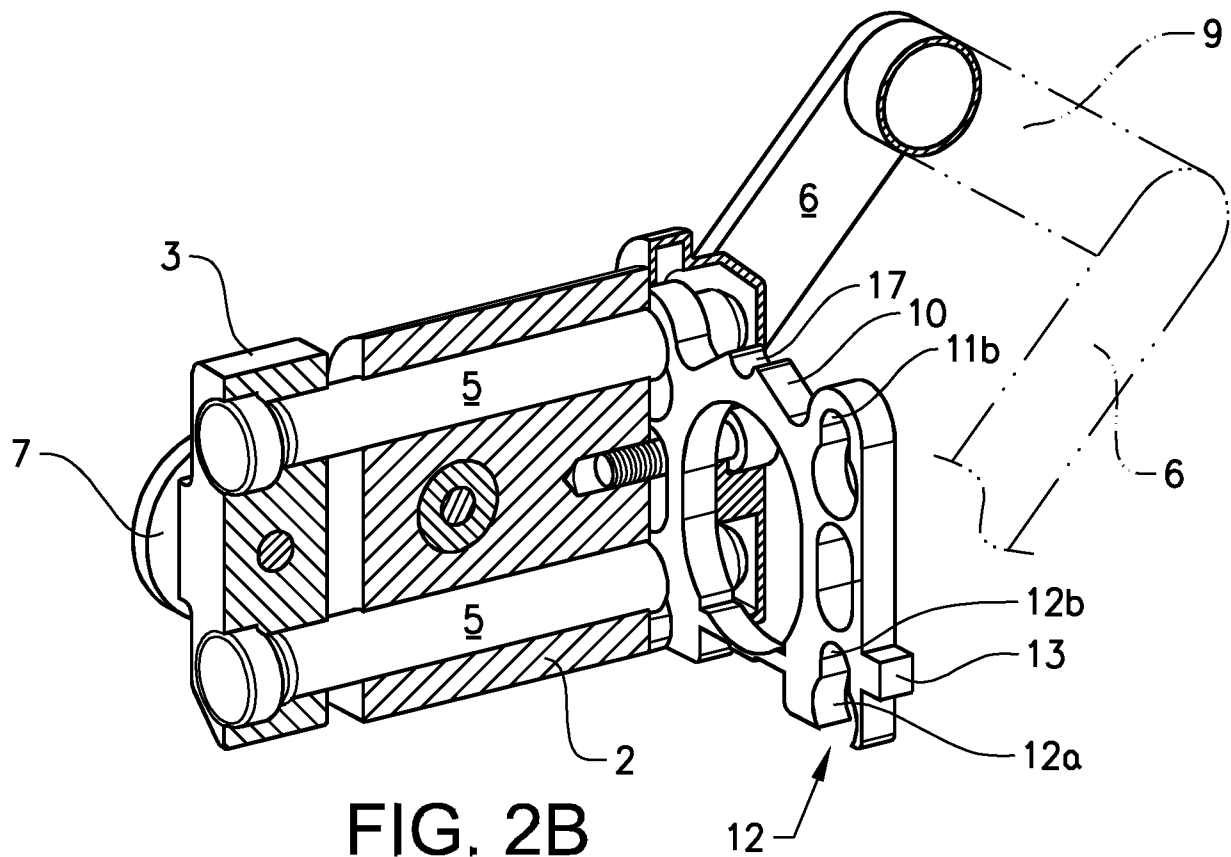


FIG. 2B

3/5

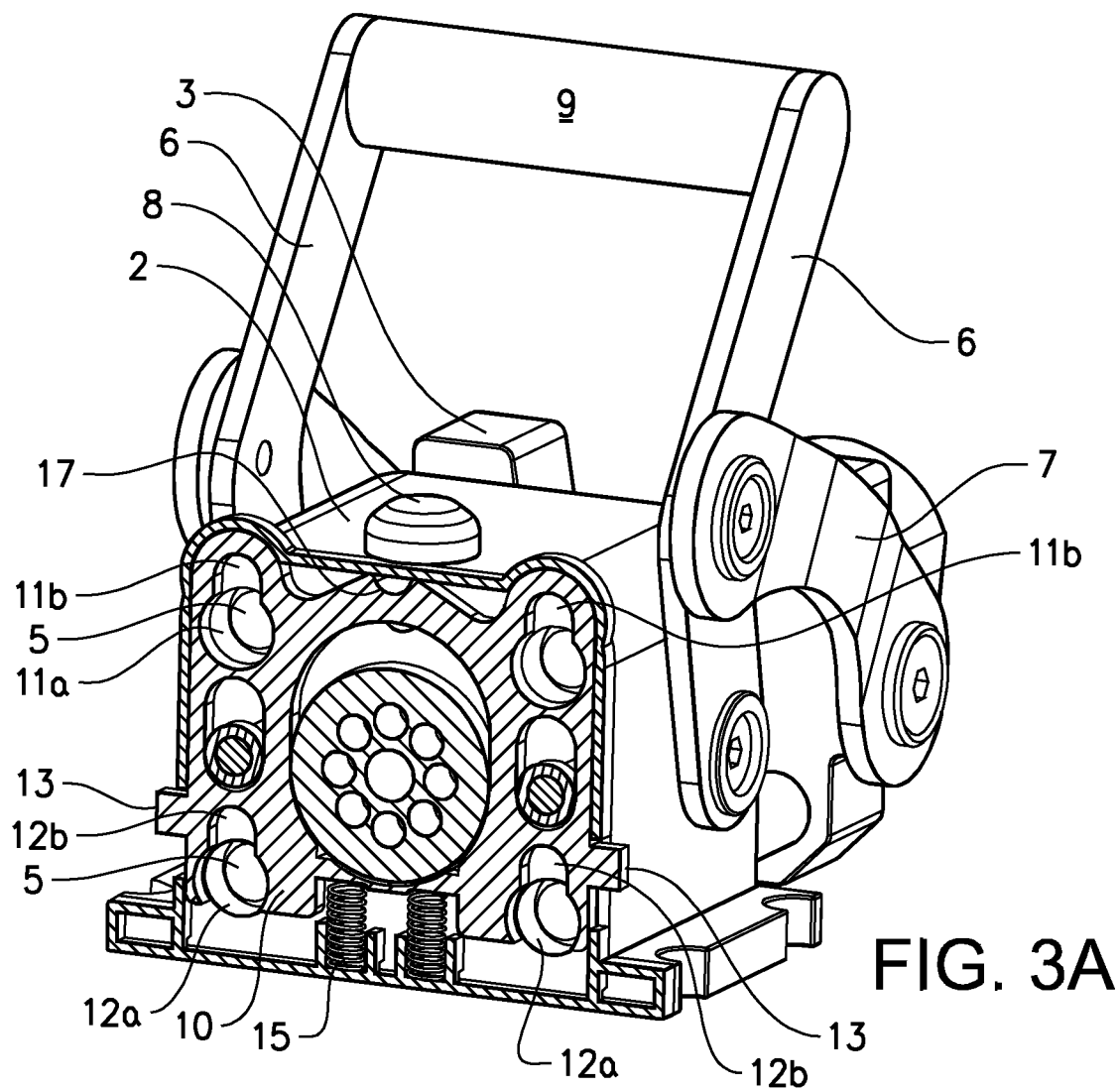


FIG. 3A

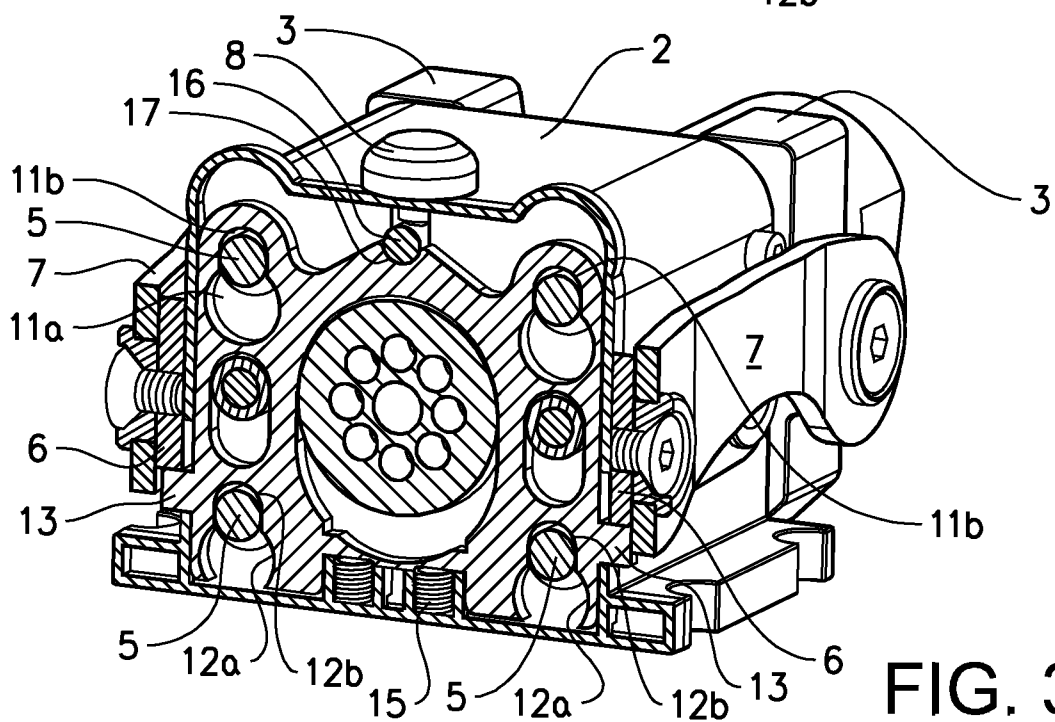
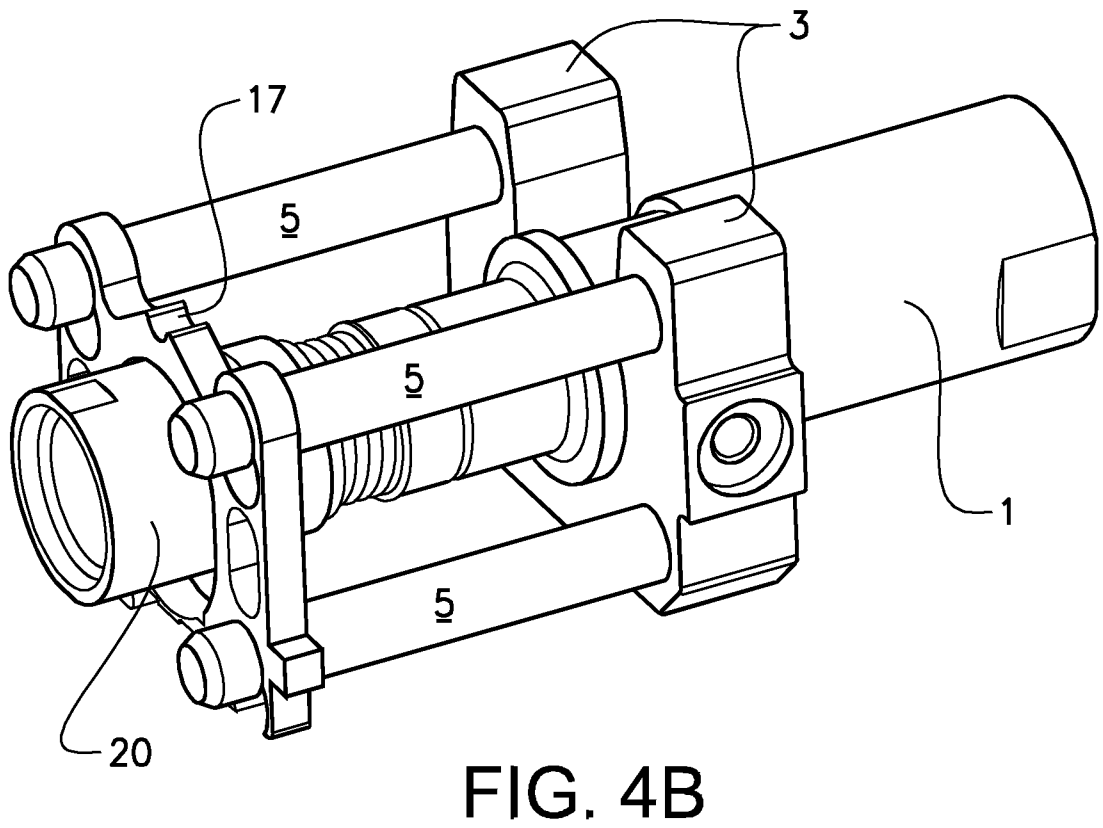
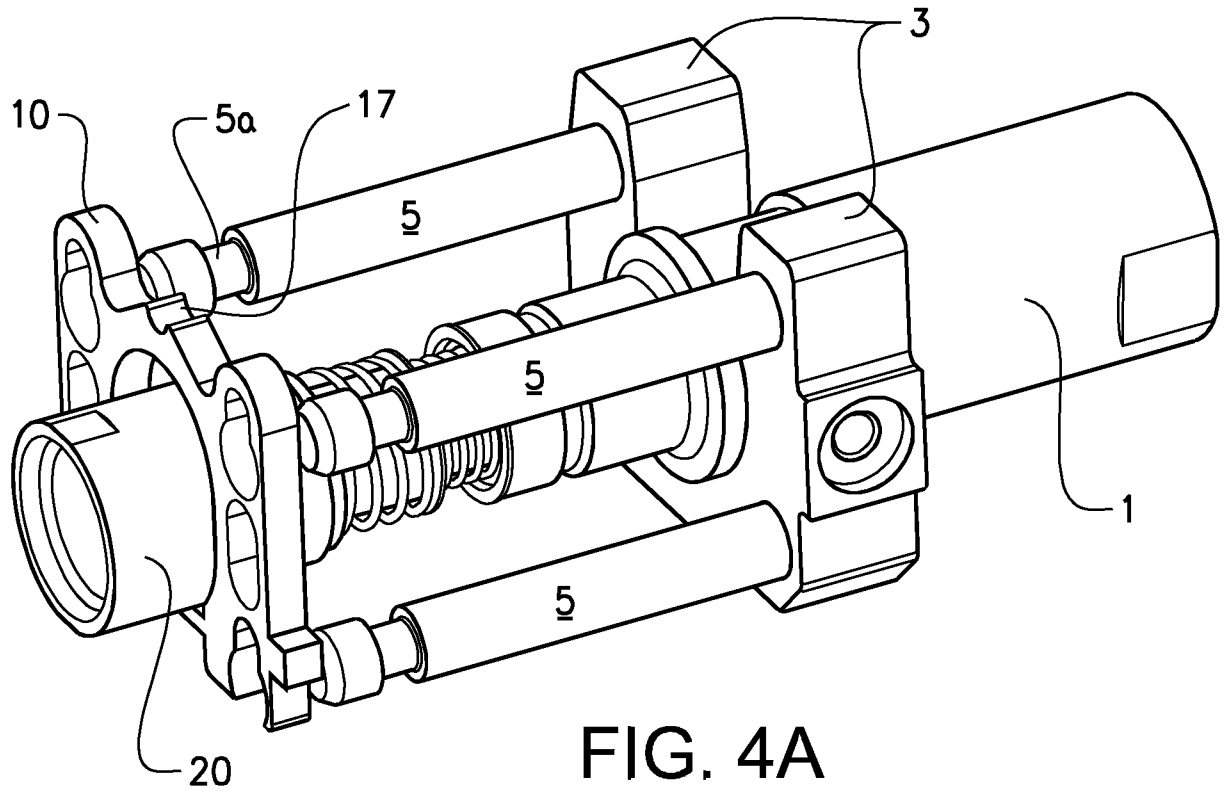


FIG. 3B

4/5



5/5

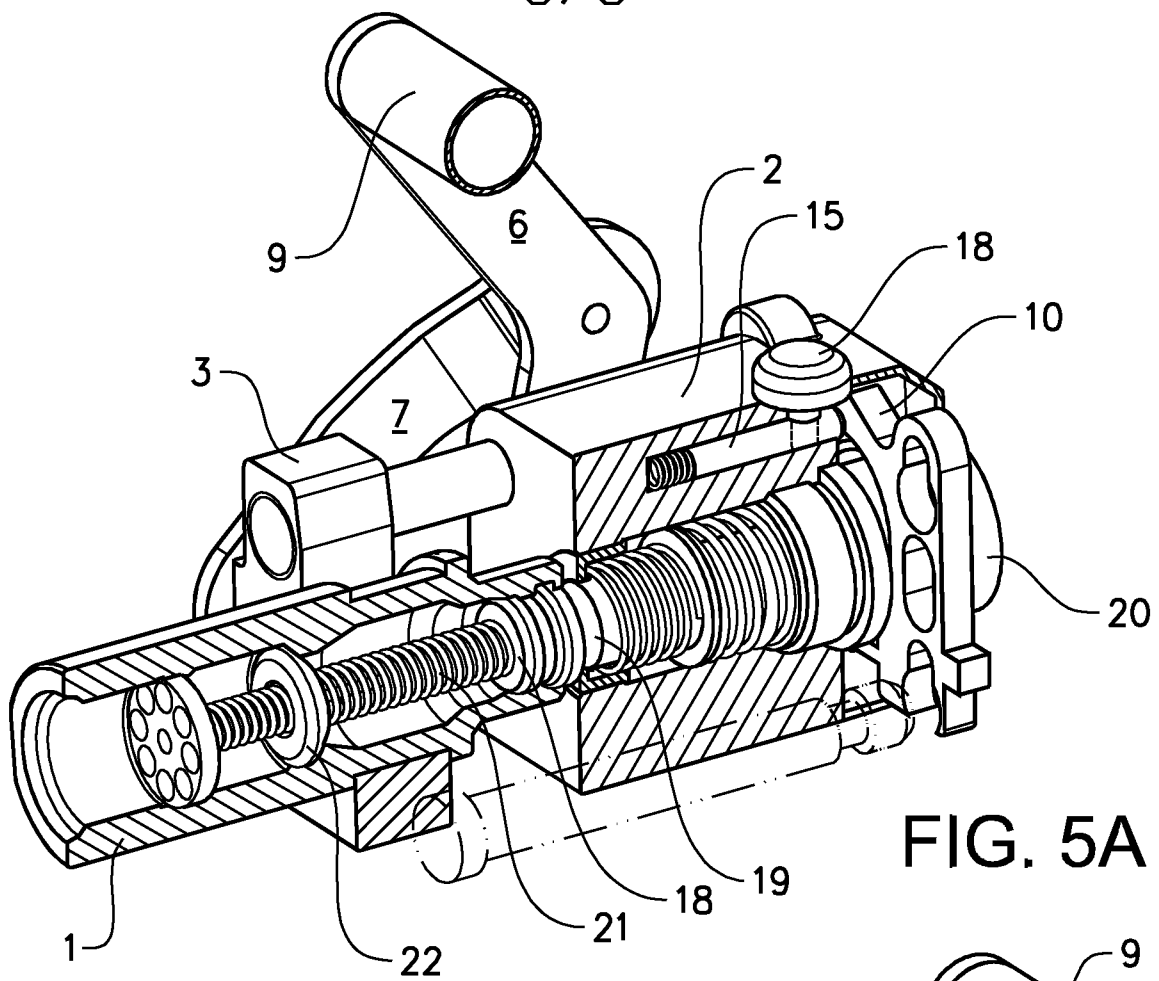


FIG. 5A

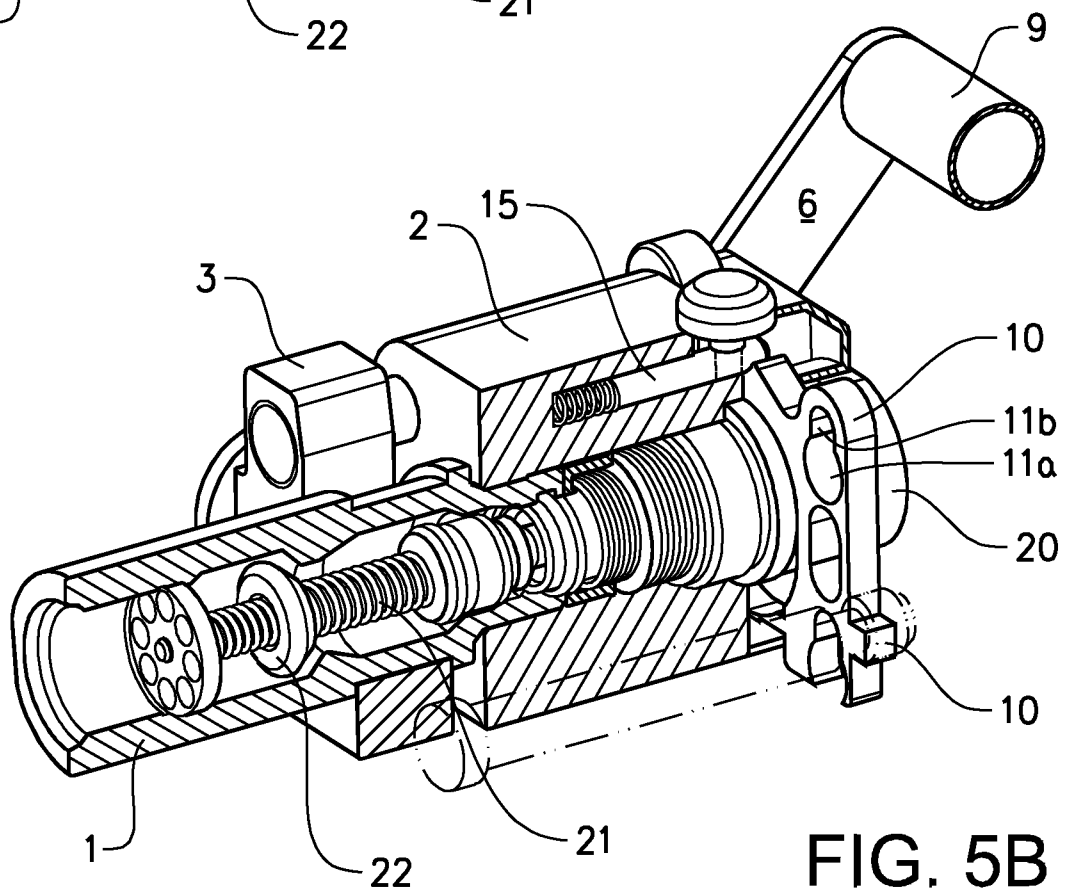


FIG. 5B

INTERNATIONAL SEARCH REPORT

International application No

PCT/EP2009/064704

A. CLASSIFICATION OF SUBJECT MATTER

INV. F16L37/14

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

F16L

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
Y	DE 10 2006 060005 A1 (TEMA MARKETING AG) 9 July 2008 (2008-07-09) abstract; figures 1,7-12 -----	1-12
Y	DE 20 2008 008692 U1 (PISTER KUGELHÄHNE GMBH) 16 October 2008 (2008-10-16) abstract; figure 1 -----	1-12
Y	GB 2 224 325 A (NATIONAL COUPLING COMPANY INCORPORATED) 2 May 1990 (1990-05-02) abstract; figures 1-5 -----	1-12
A	DE 42 22 193 A1 (FASTER S.R.L.) 14 January 1993 (1993-01-14) the whole document -----	1-12
A	FR 2 882 804 A (MAISONNEUVE) 8 September 2006 (2006-09-08) the whole document -----	1-12



Further documents are listed in the continuation of Box C.



See patent family annex.

* Special categories of cited documents :

- *A* document defining the general state of the art which is not considered to be of particular relevance
- *E* earlier document but published on or after the international filing date
- *L* document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)
- *O* document referring to an oral disclosure, use, exhibition or other means
- *P* document published prior to the international filing date but later than the priority date claimed

- *T* later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention
- *X* document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone
- *Y* document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art.
- *&* document member of the same patent family

Date of the actual completion of the international search

25 January 2010

Date of mailing of the international search report

02/02/2010

Name and mailing address of the ISA/

European Patent Office, P.B. 5818 Patentlaan 2
NL - 2280 HV Rijswijk
Tel. (+31-70) 340-2040,
Fax: (+31-70) 340-3016

Authorized officer

Kepka, Maciek

INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No

PCT/EP2009/064704

Patent document cited in search report	Publication date	Patent family member(s)	Publication date
DE 102006060005 A1	25-10-2007	NONE	
DE 202008008692 U1	11-09-2008	NONE	
GB 2224325	A	02-05-1990	
		AU 632766 B2	14-01-1993
		AU 4350989 A	03-05-1990
		BR 8905483 A	29-05-1990
		DE 3935950 A1	10-05-1990
		NO 894288 A	30-04-1990
		US 4915419 A	10-04-1990
DE 4222193	A1	14-01-1993	
		AT 130919 T	15-12-1995
		EP 0522493 A1	13-01-1993
		US 5316347 A	31-05-1994
FR 2882804	A	08-09-2006	NONE