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(54) **FAN ASSEMBLY AND METHOD**

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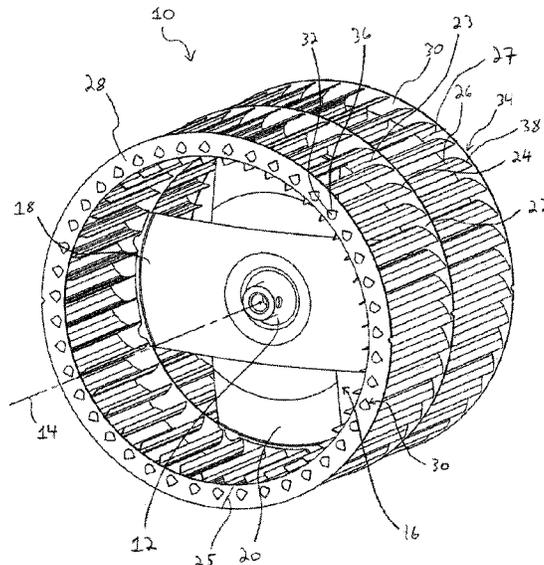
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(57) **ABSTRACT**

In one aspect, a fan assembly is provided that includes a plurality of elongated metallic fan blades, a first metallic end ring, a second metallic end ring, and a metallic hub support connected to the fan blades intermediate the first and second end rings. The fan blades include straight inlet edges radially inward from first and second radially inner edges of the first and second metallic end rings so that inlet portions of the blades extend radially inward from the first and second metallic end rings. In another aspect, a fan assembly is provided that includes a hub support having a hub strip and a hub ring. The hub strip has an elongate configuration with an attachment portion connected to the hub ring at one end of the hub strip and another attachment portion connected to the hub ring at an opposite end of the hub strip.

23 Claims, 16 Drawing Sheets



Related U.S. Application Data

continuation of application No. 13/022,250, filed on Feb. 7, 2011, now Pat. No. 8,881,396.

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F01D 5/14 (2006.01)
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- (52) **U.S. Cl.**
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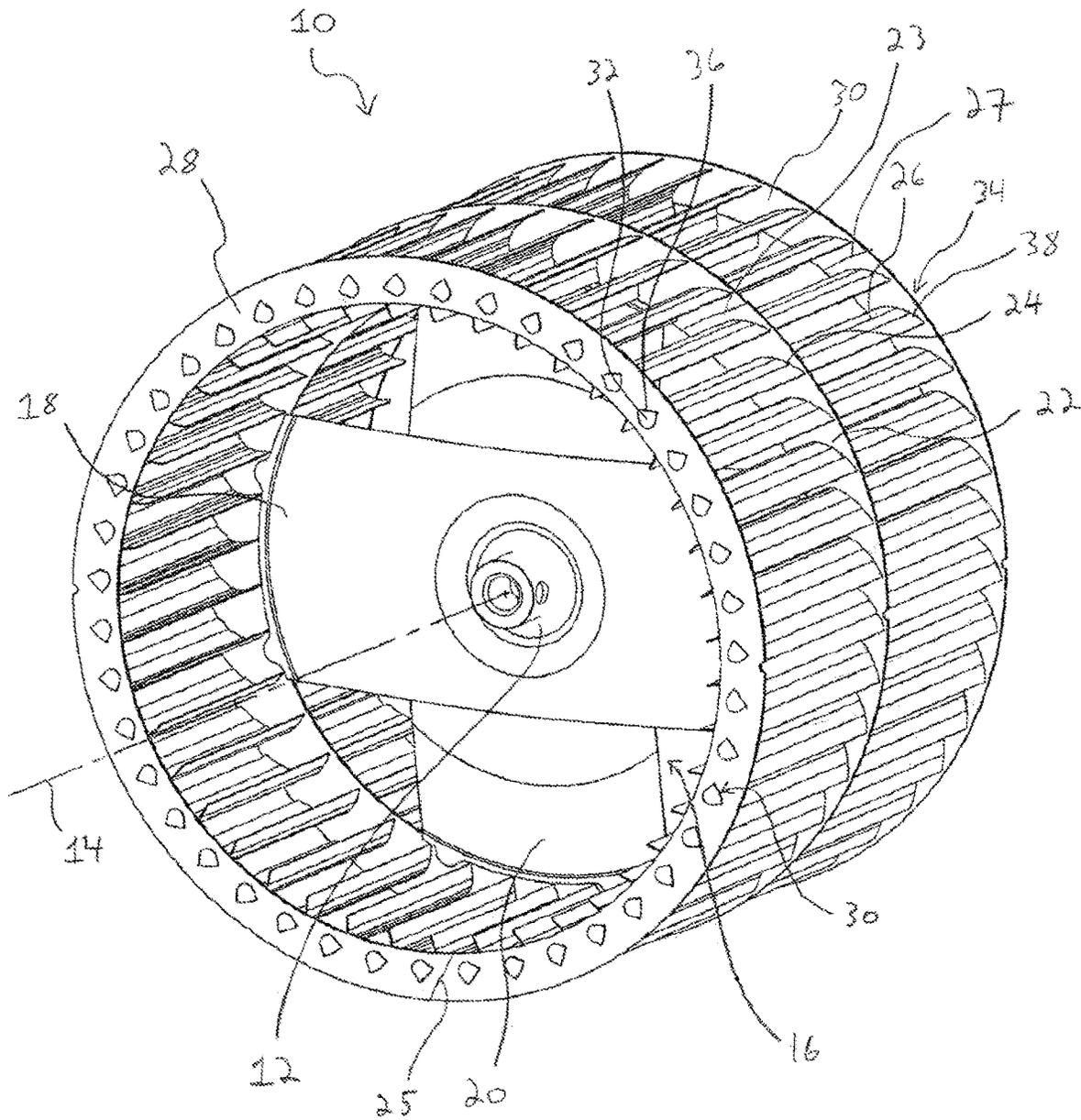


FIG. 1

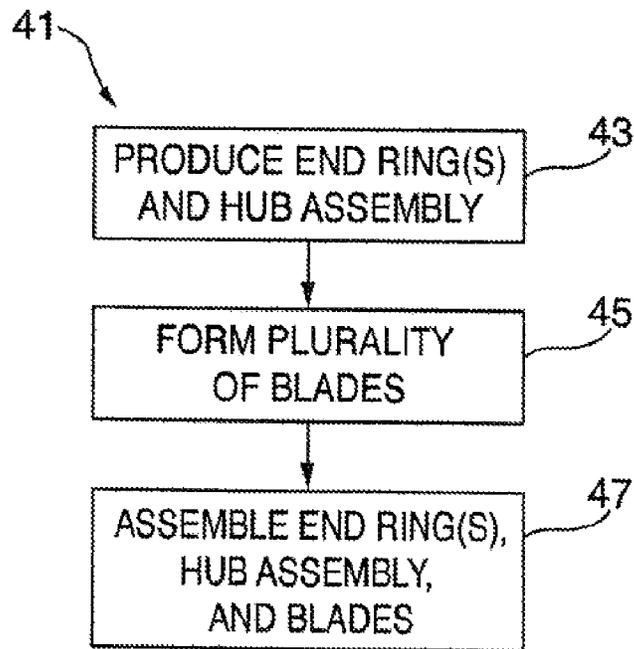
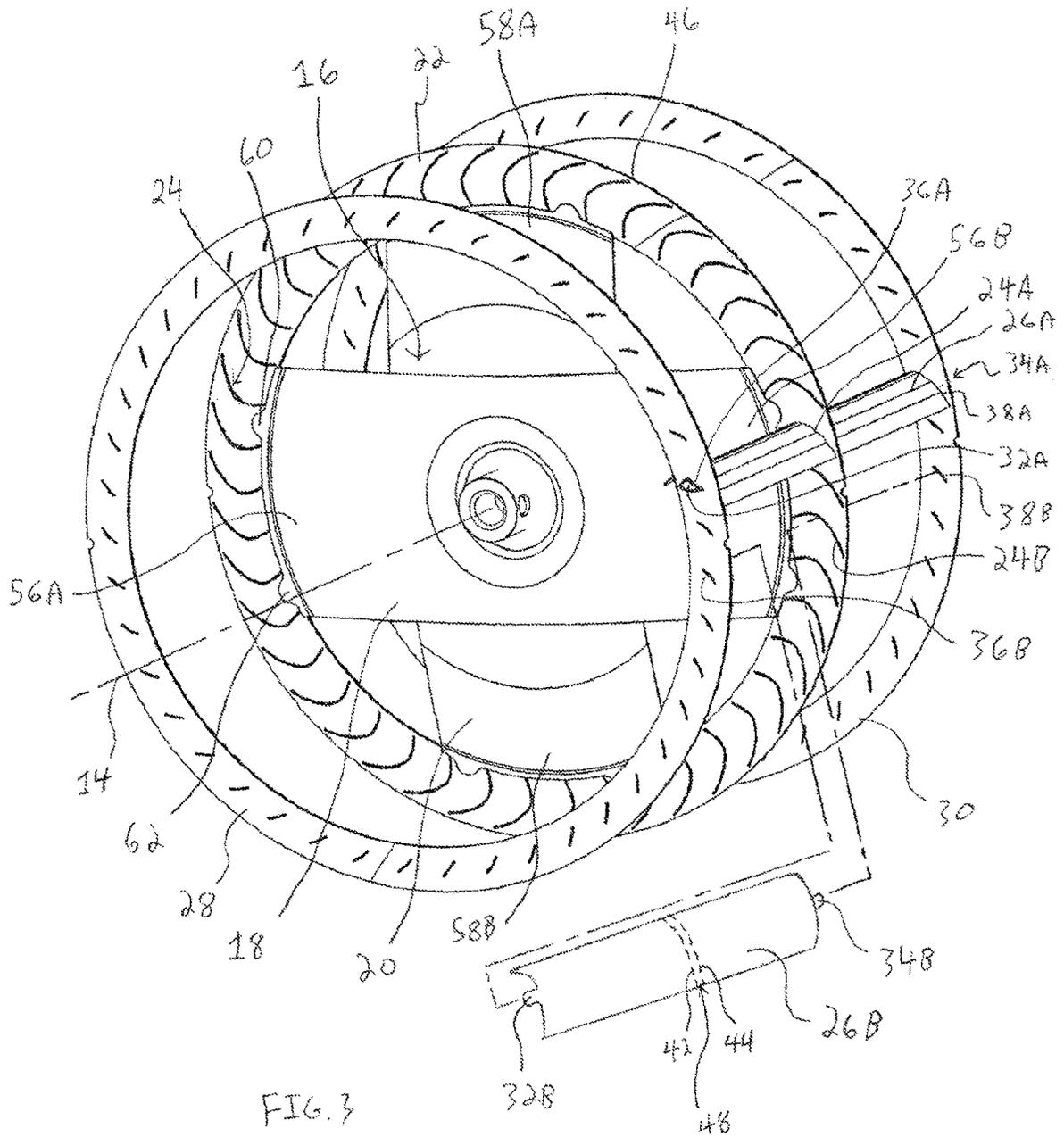


FIG. 2



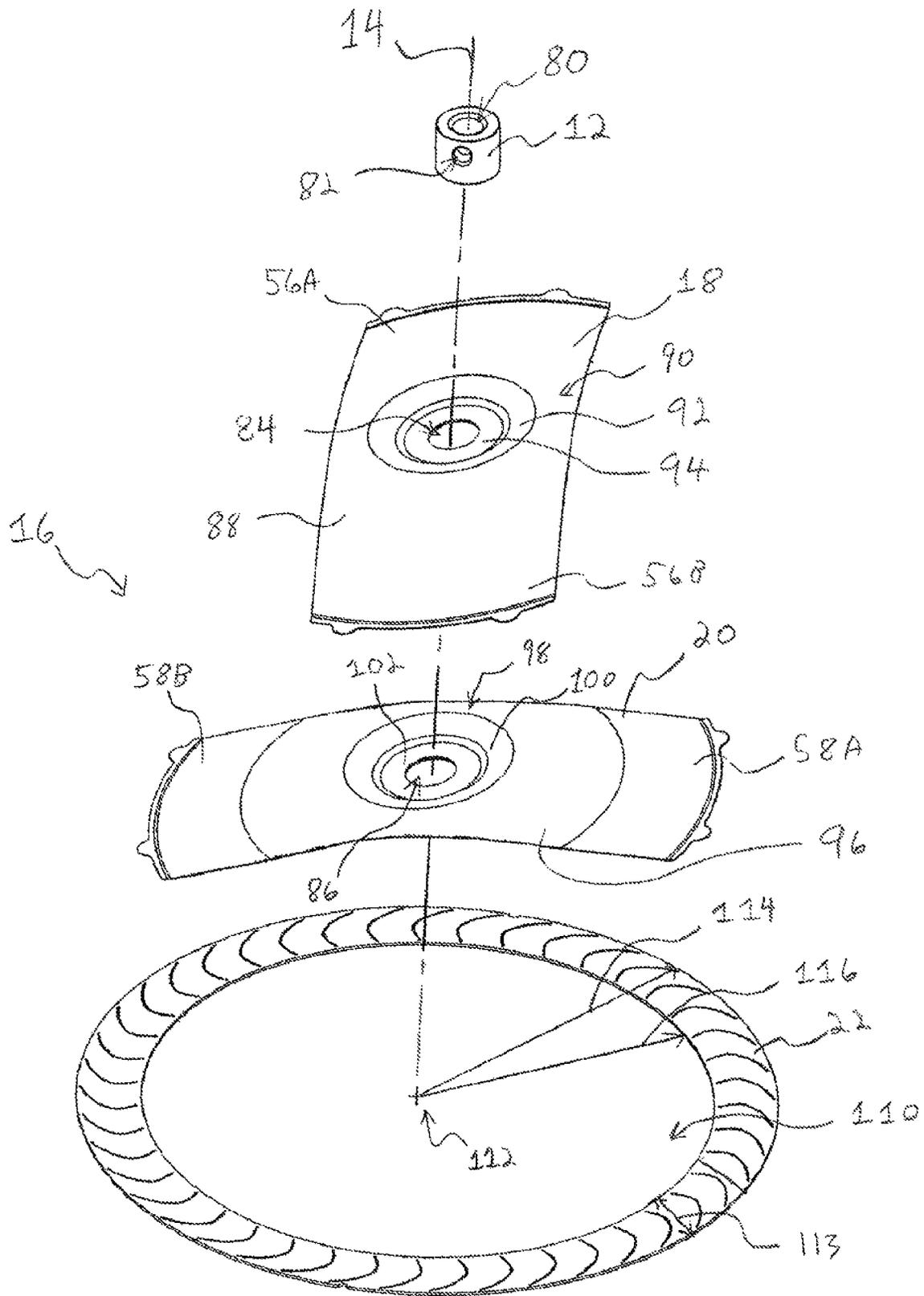


FIG. 4

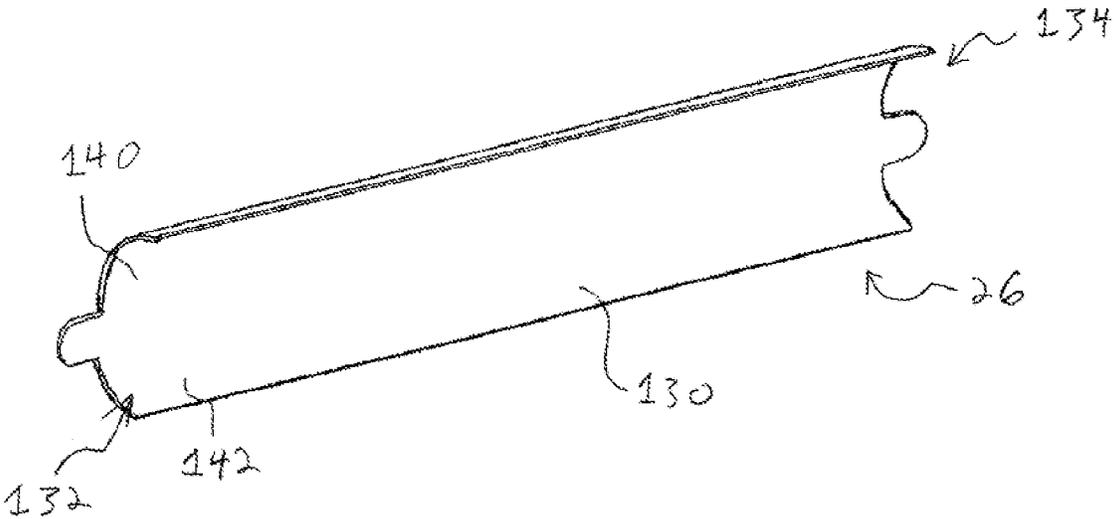


FIG. 5

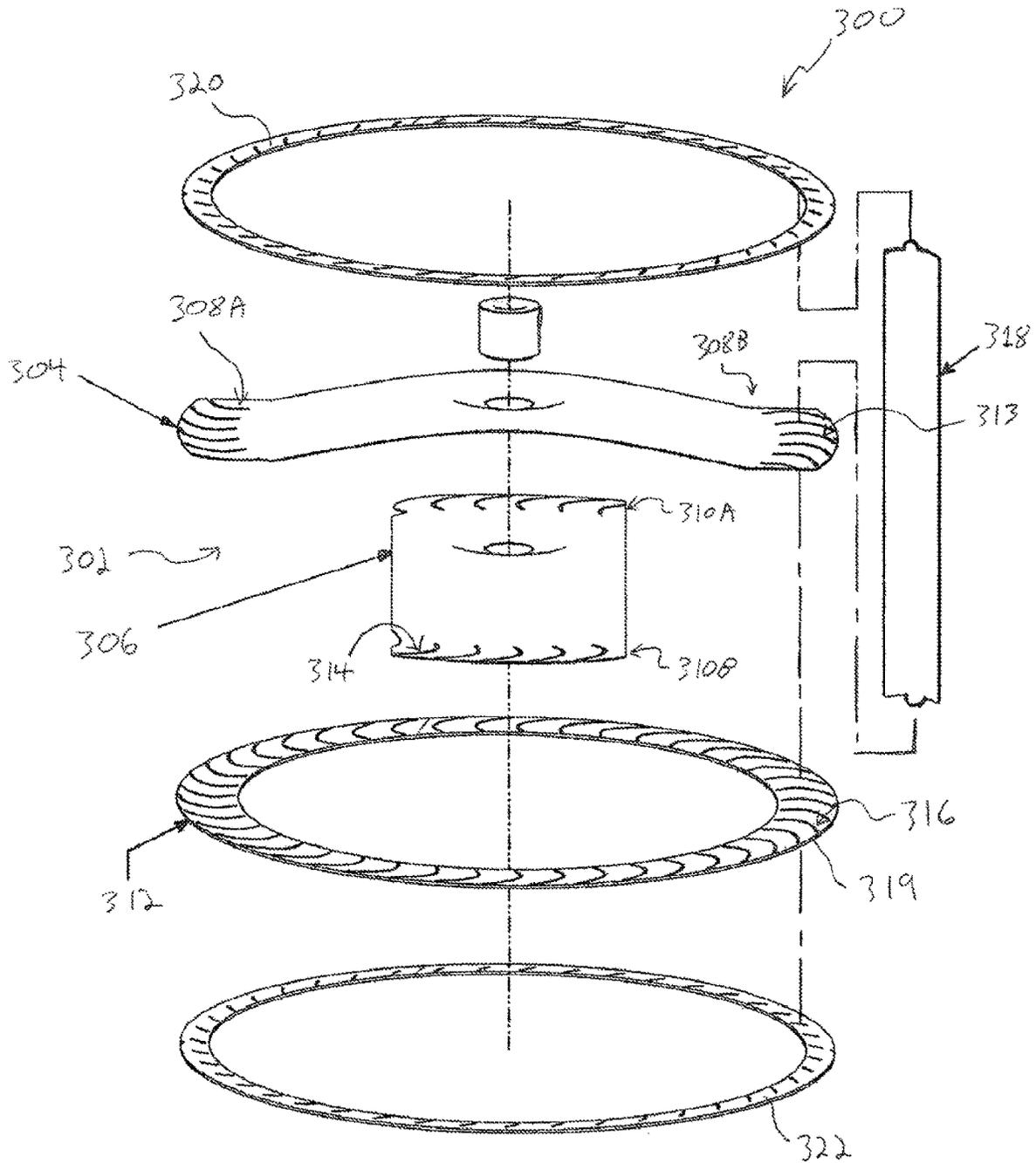


FIG. 7

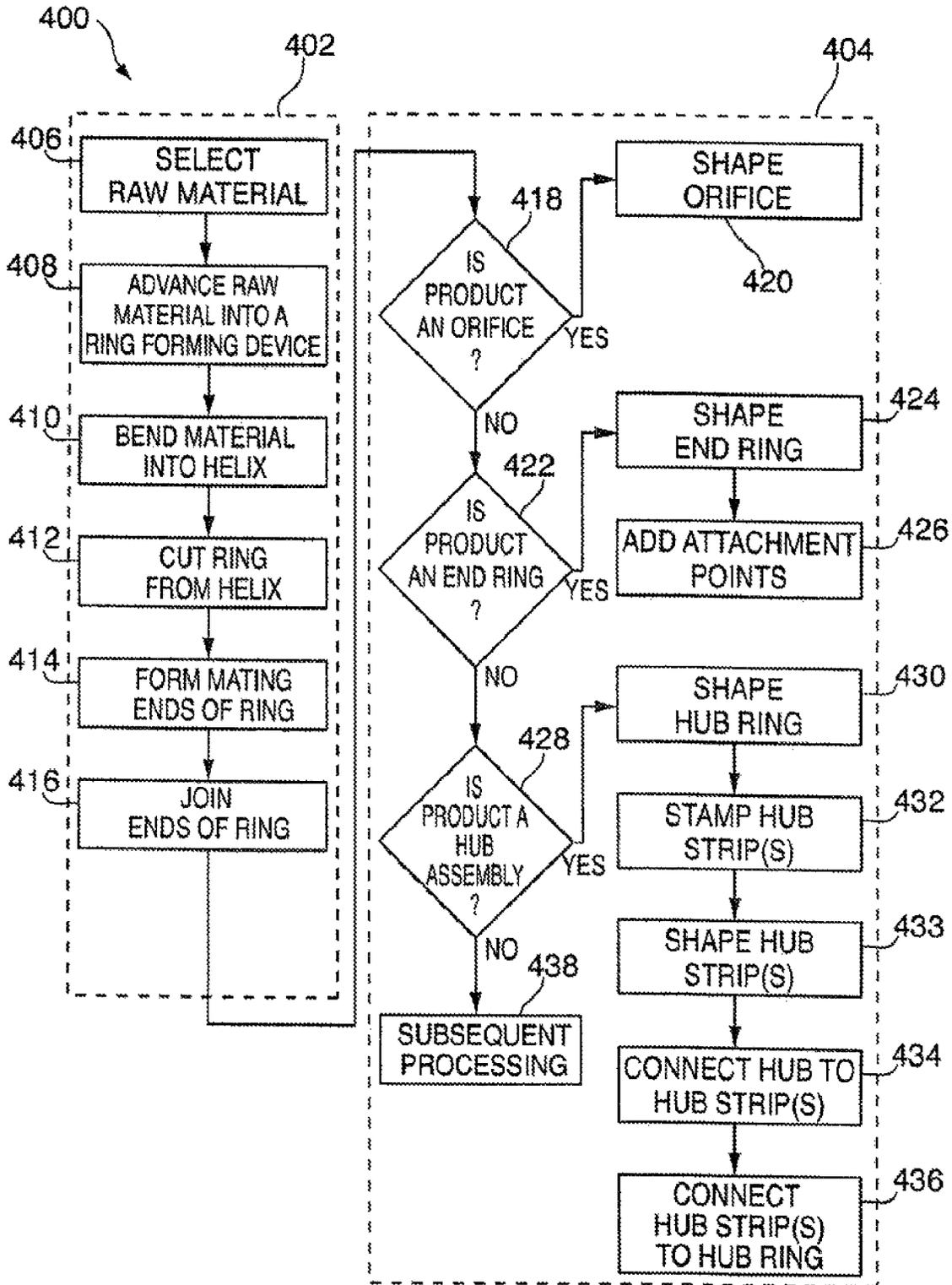
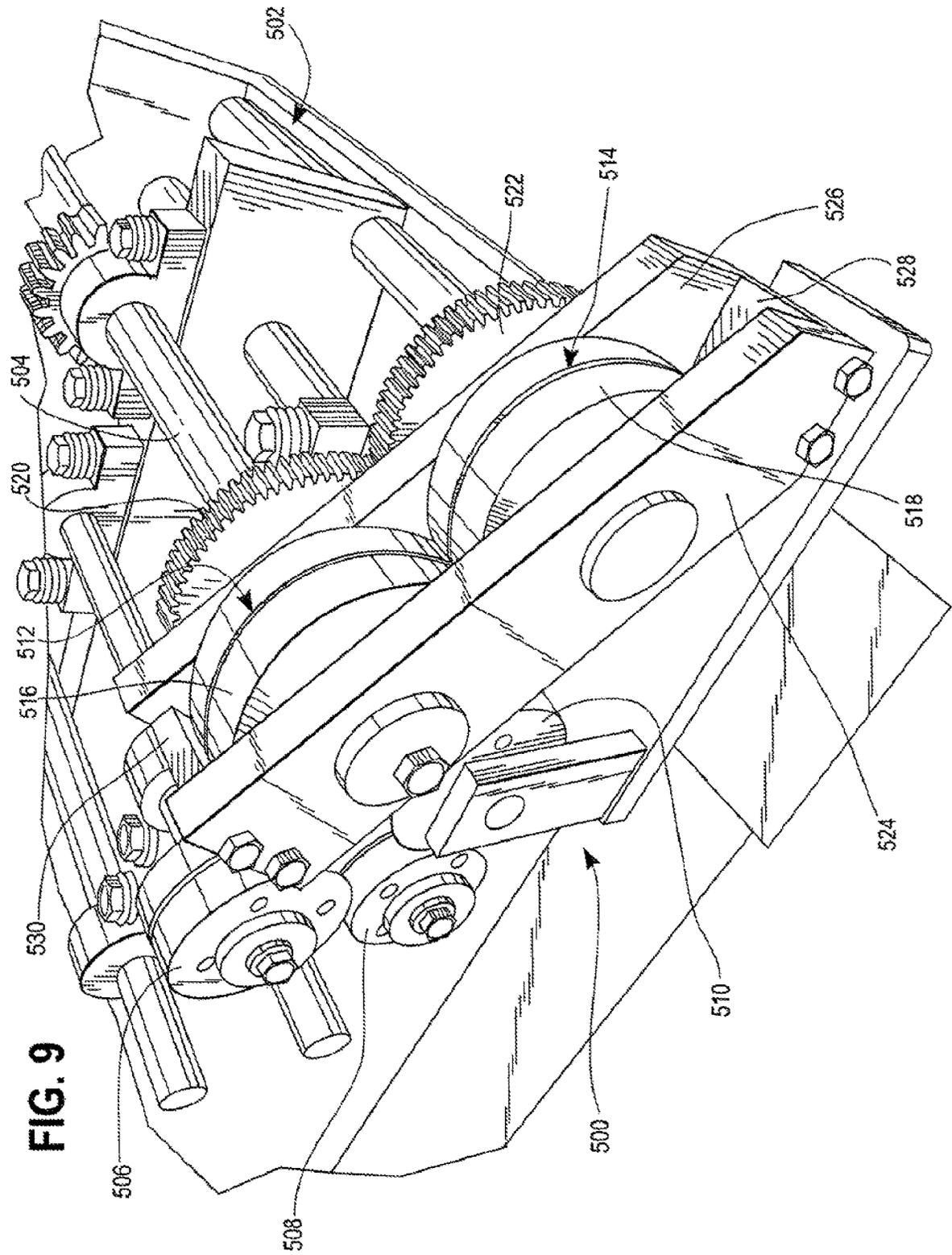


FIG. 8



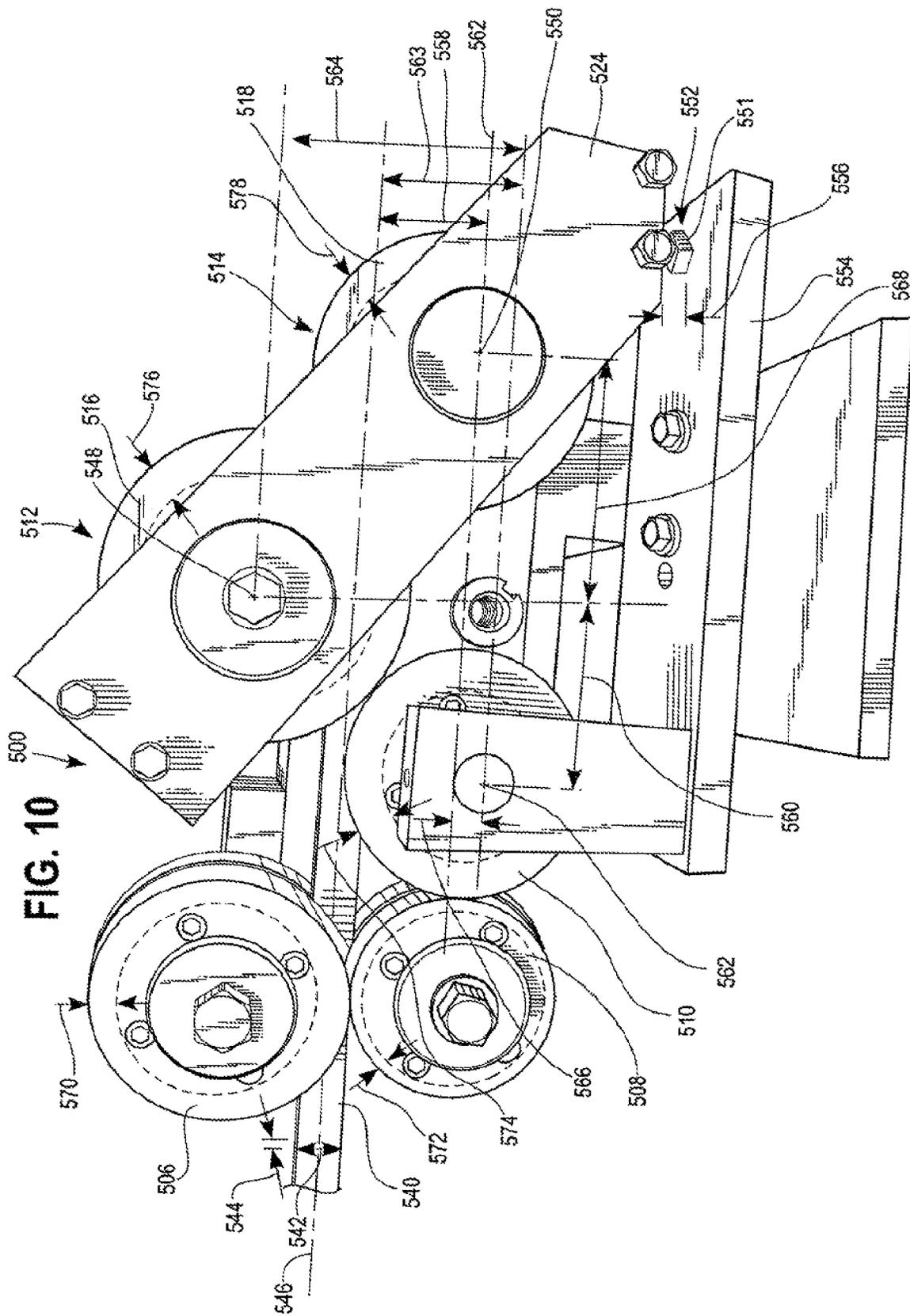


FIG. 10

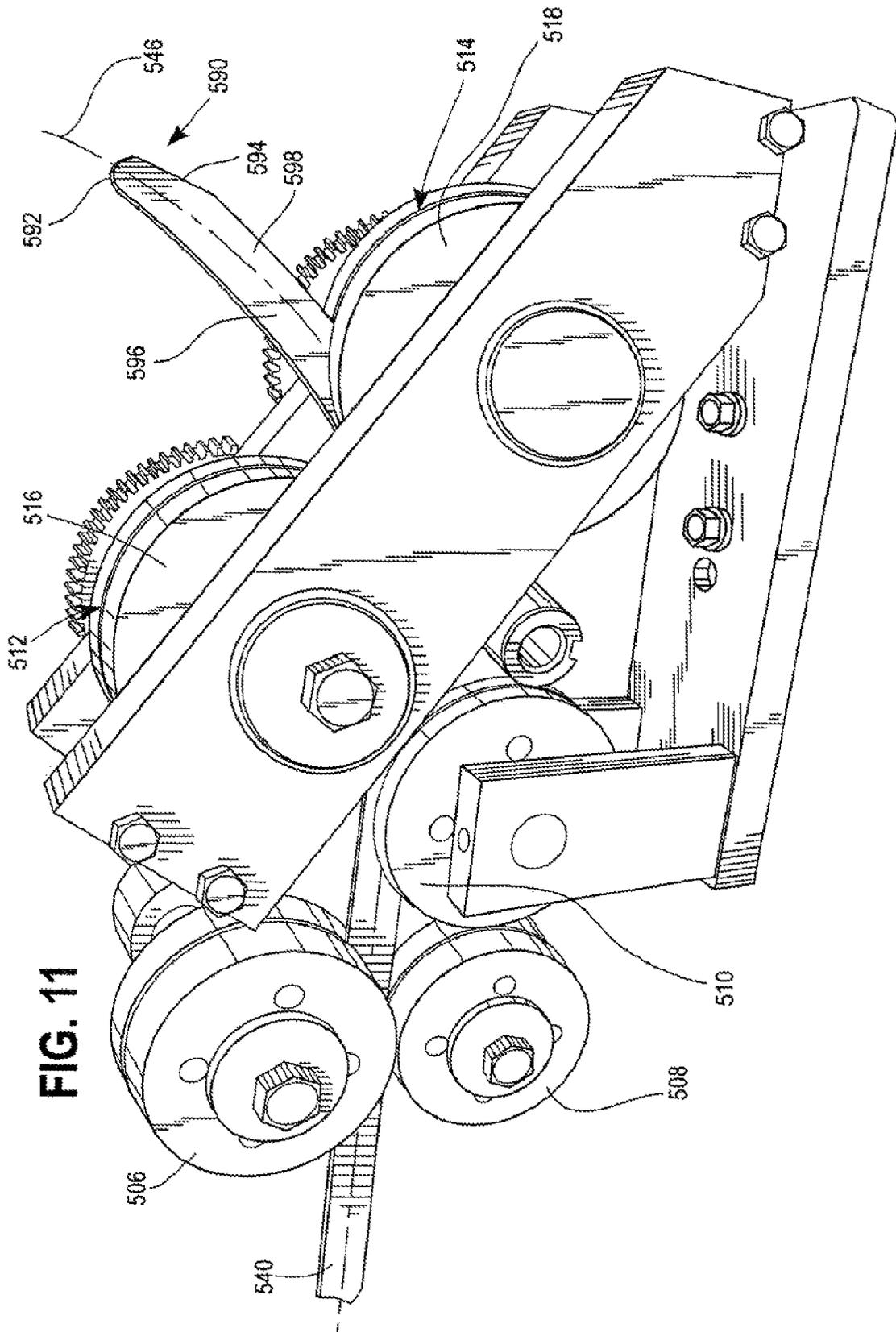
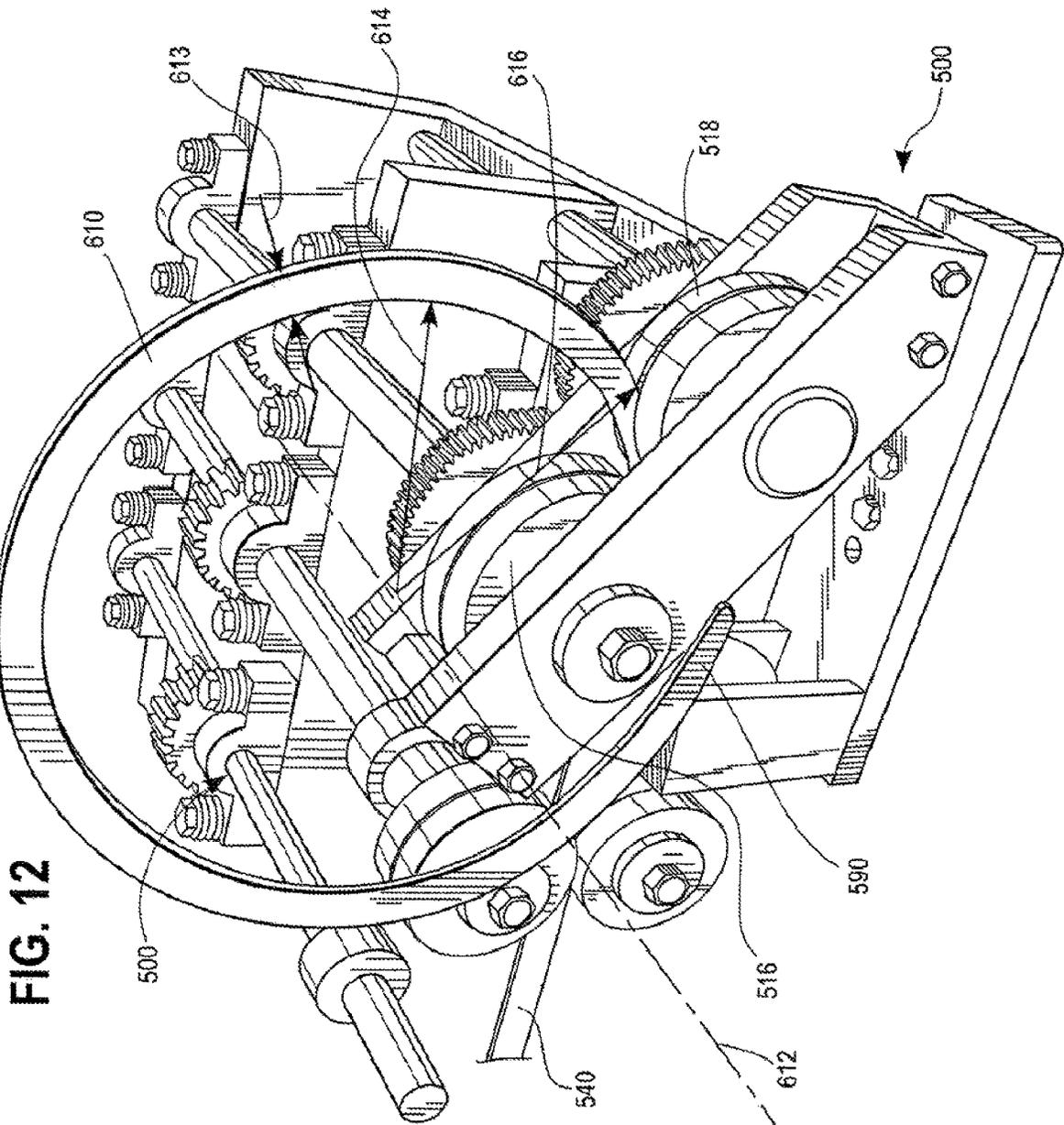


FIG. 11



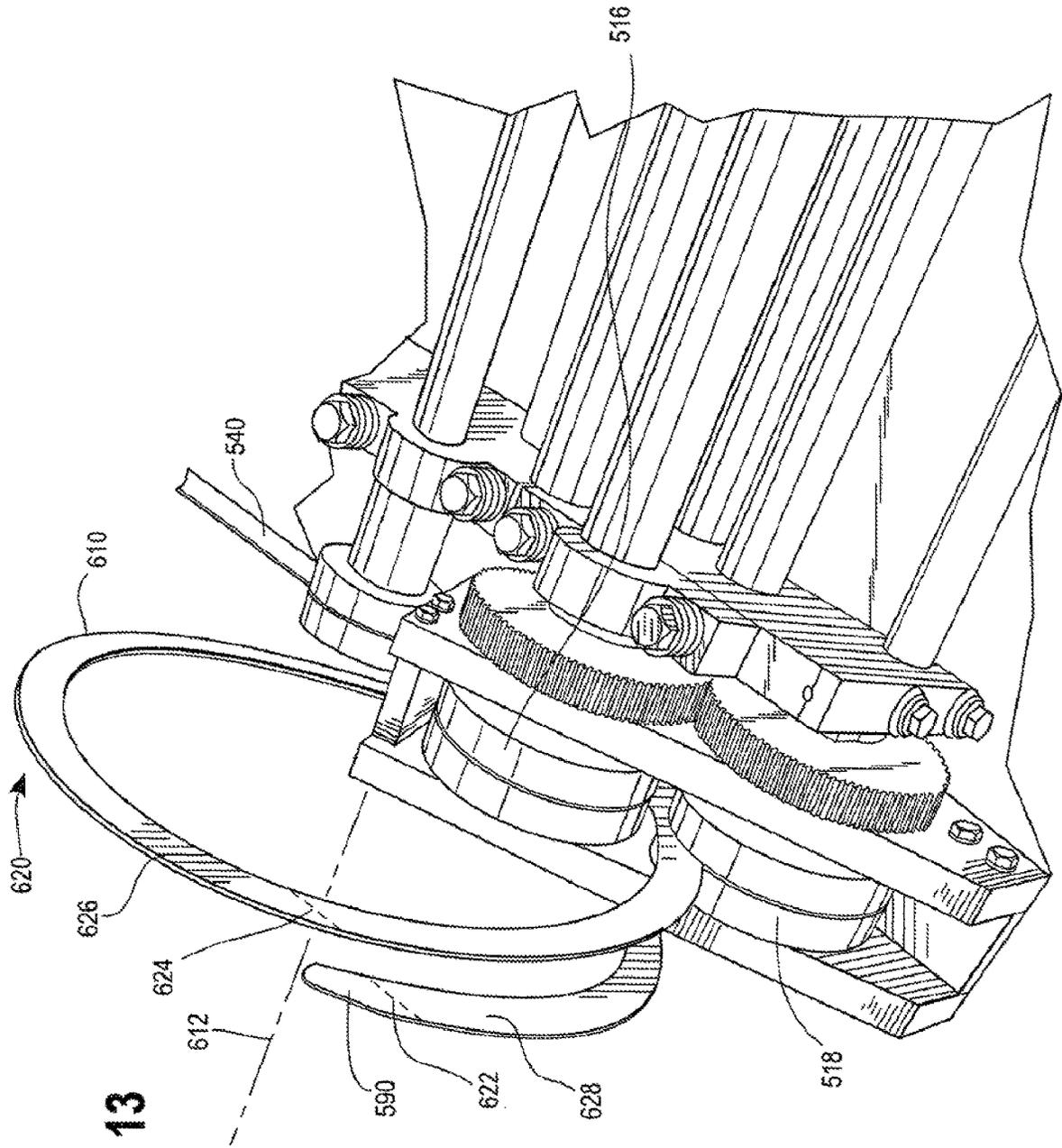


FIG. 13

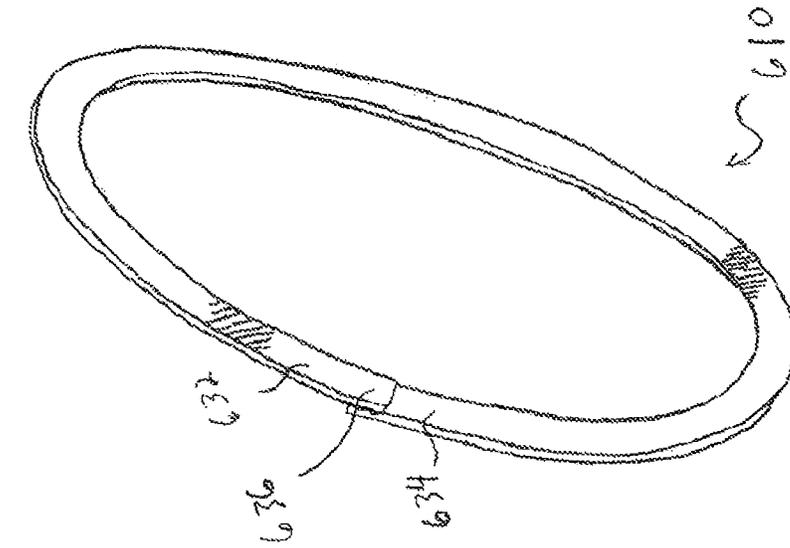


FIG. 13A

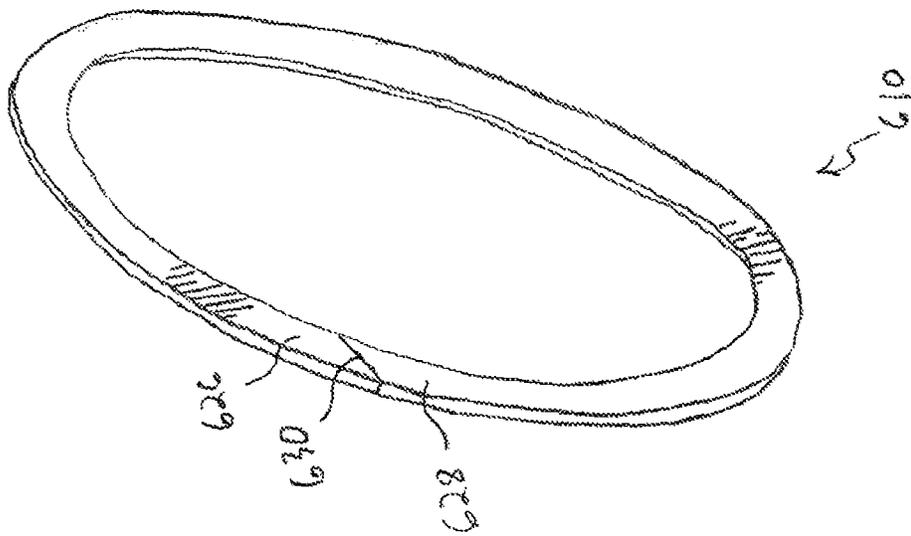


FIG. 13B

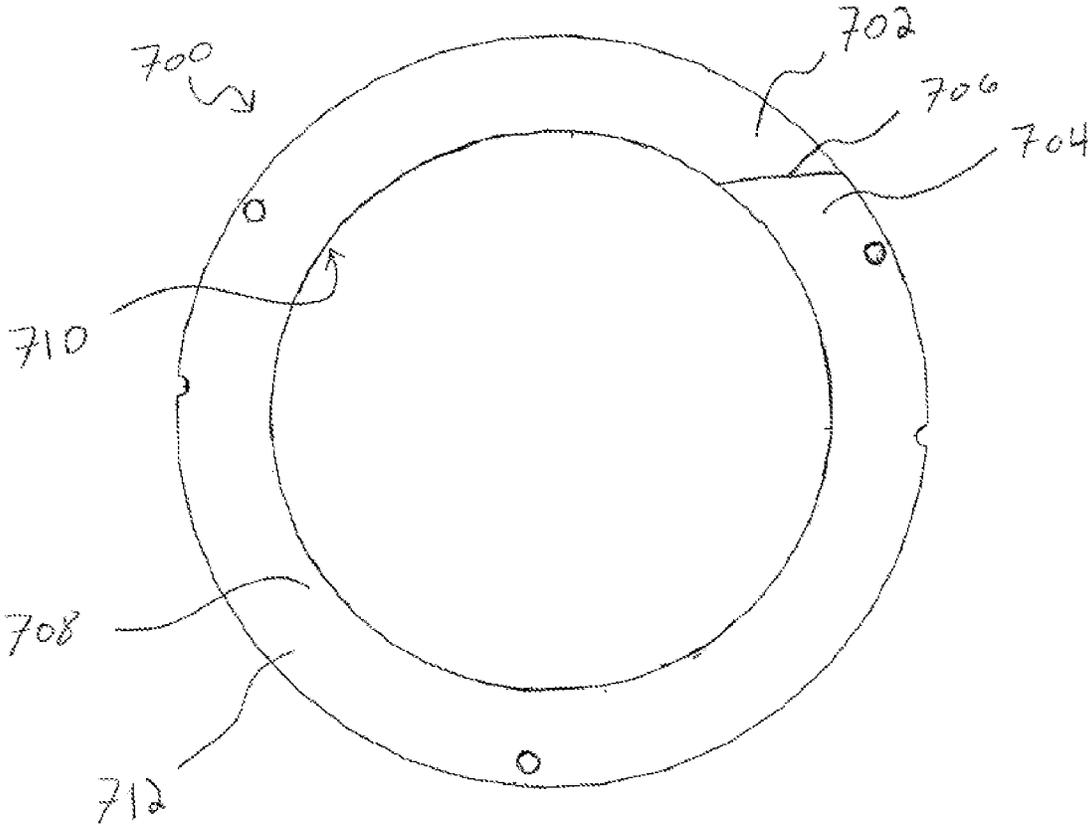


FIG. 14

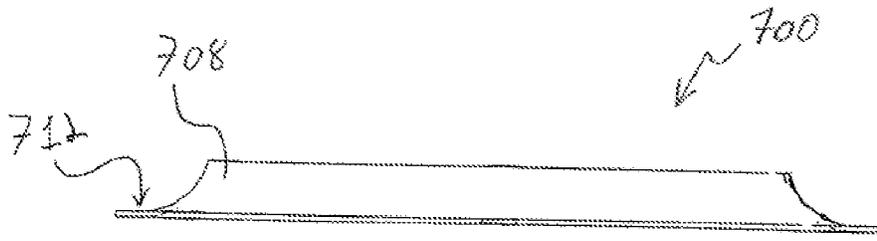


FIG. 15

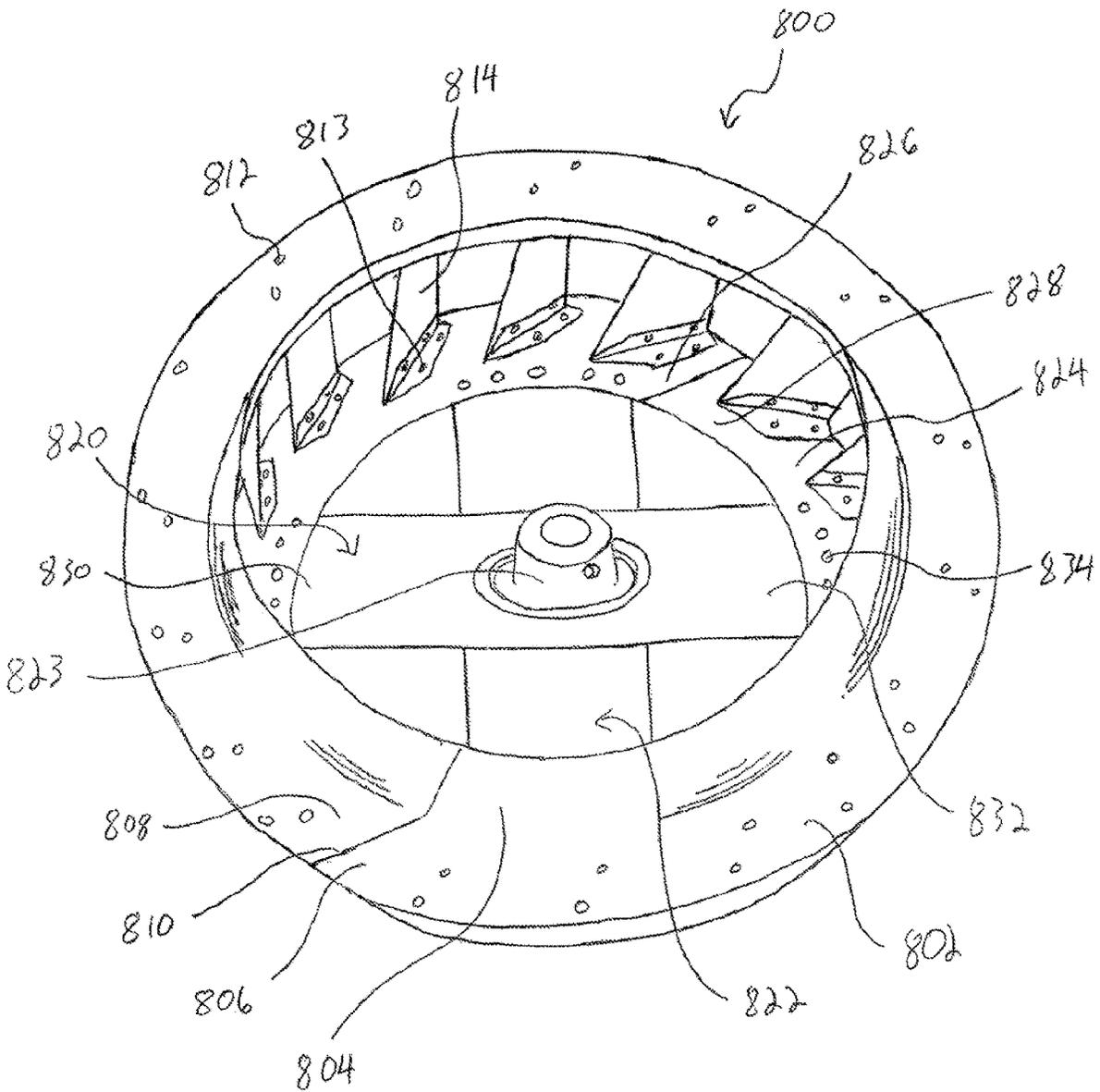


FIG. 16

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FAN ASSEMBLY AND METHOD**CROSS-REFERENCE TO RELATED APPLICATIONS**

This application is a continuation of U.S. patent application Ser. No. 14/517,212, filed Oct. 17, 2014, now U.S. Pat. No. 10,125,784, which is a continuation of U.S. patent application Ser. No. 13/022,250, filed Feb. 7, 2011, now U.S. Pat. No. 8,881,396, which are incorporated herein by reference.

FIELD

The field relates to fans and, more particularly, to fan assemblies for handling airflow.

BACKGROUND

Centrifugal or forward curved blower wheels may generally comprise a plurality of blades arranged in the form of a cylinder around a solid, central hub disc. During manufacture of a centrifugal blower wheel, a stamping press may be used to form the plurality of blades from a metal sheet. The sheet is gathered between the ends of adjacent blades to reduce the distance between the blades. The sheet, with the material gathered between the blades, is then shaped into a cylinder about the solid, central hub disc. Because the blades are stamped from a single metal sheet, the number of blades and the chords of the blades are limited by the size of the blades and the size of the sheet.

The solid, central hub disc of the centrifugal blower wheel may be formed by stamping a pair of discs from respective rectangular pieces of sheet metal, removing the scrap material for recycling, and joining the discs together. A centrifugal blower wheel may also have one or more end rings deformed onto ends of the plurality of blades to fix the blades in the cylindrical arrangement about the central hub disc. Like the central hub disc, the end rings may be formed by stamping circular rings from rectangular pieces of sheet metal and removing the scrap material for recycling. Stamping the one or more end rings generates additional scrap because each end ring has a central opening formed by removing a circular disc from the center of the end ring. In one approach, the scrap material removed from an end ring to form the central opening can be used as a solid, central hub disc for the centrifugal blower wheel. Although scrap material produced during manufacture of the centrifugal blower wheel may be recycled, the value of the recycled material may not compensate for the costs incurred in cutting, storing, and handling the scrap material.

Mixed flow fans, backward inclined wheels, backward curved wheels, and other centrifugal wheels may have a back disc and an end ring that are both stamped from rectangular pieces of metal, like the central hub disc and end ring(s) of centrifugal blower wheels. Because the back disc and the end ring have a generally circular shape, stamping the circular shapes from rectangular pieces of sheet metal produces scrap in the form of the corner portions of the sheet metal and a circular disc from the center of the end ring. Similarly, traditional approaches for producing a venturi or orifice for a fan assembly or a fan housing include stamping a generally ring-shaped orifice from a rectangular piece of metal. Even if the scrap circular disc from the end ring or orifice is used to produce a back disc, the corner portions of

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the sheet located outside of the end ring or orifice would still constitute a large amount of scrap material.

BRIEF DESCRIPTION OF THE DRAWINGS

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FIG. 1 is a perspective view of a fan assembly;

FIG. 2 is a flow diagram of a method of manufacturing the fan assembly of FIG. 1.

FIG. 3 is a perspective view of the fan assembly of FIG. 1 with all but two fan blades removed;

FIG. 4 is an exploded perspective view of a hub assembly of the fan assembly of FIG. 1;

FIG. 5 is a perspective view of a fan blade of the fan assembly of FIG. 1;

FIG. 6 is a side elevational view of the fan blade of FIG. 5;

FIG. 7 is an exploded perspective view of another fan assembly with only one blade shown for clarity;

FIG. 8 is a flow diagram of a method of producing parts of fan assemblies and other components;

FIGS. 9-13 are perspective views of a ring forming sub-method of the method of FIG. 8;

FIGS. 13A and 13B are perspective views of rings formed using the sub-method of FIGS. 9-13;

FIG. 14 is a top plan view of an orifice;

FIG. 15 is a side elevational view of the orifice of FIG. 14; and

FIG. 16 is a perspective view of another fan assembly.

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

In one aspect, a method is provided that minimizes the amount of scrap material produced during manufacture of blower wheel or fan assemblies and related components. Rather than stamp circular parts from rectangular sheets of metal, as in traditional approaches, a narrow strip of material is advanced from a coil and bent along a longitudinal edge of the strip to form a helix. An open annular member is cut from the helix and trimmed as needed before the ends of the open annular member are joined together to form a rigid, annular member. The rigid annular member may be used in place of traditional orifices and fan assembly end rings, to name a few applications. Further, by joining the rigid, annular member with one or more hub strips to produce a hub assembly, the rigid annular member may be used in place of central or end discs of traditional fan assemblies.

In another approach, a fan assembly is provided that minimizes the amount of scrap material produced during manufacture of the fan assembly. More specifically, the fan assembly does not utilize a solid disc traditionally used as a center or end disc for blower wheels, mixed flow fans, or other centrifugal air moving devices. Instead, the fan assembly has a hub assembly comprising a hub ring and one or more hub strips. Each hub strip has an attachment portion connected to the hub ring and a body portion extending radially inward from the attachment portion. The one or more hub strips are configured to connect directly or indirectly to a motive source, such as an electric motor, a diesel or gasoline engine, or a turbine, using known mechanical linkages. In one approach, the one or more hub strips are connected to a hub that is fixed to a driveshaft of an electric motor. In another approach, the one or more hub strips have a mounting point or feature for attachment to a shaft or clamp. As is apparent, the hub assembly transmits rotation from the motive source to the fan assembly without the use of a solid disc, as in traditional fan assemblies.

Turning to the Figures, FIG. 1 illustrates a fan assembly 10 that produces a minimal amount of scrap material during production. The fan assembly 10 has a hub 12 for engaging a motor drive shaft (not shown) and transmitting rotation of the drive shaft to the fan assembly 10 such that the fan assembly 10 rotates about an axis of rotation 14. The hub 12 is part of a hub assembly 16 that includes the hub 12, one or more hub strips 18, 20, and a hub ring 22. The hub strips 18, 20 connect the hub 12 to the hub ring 22. In an alternative embodiment, the hub 12 is integrally formed with the hub strips 18, 20.

The hub ring 22 has a plurality of attachment points, such as slots 24, for engaging a plurality of fan blades 26. Each fan blade 26 extends between a pair of end rings 28, 30 and through an associated slot 24 in the hub ring 22. The hub ring 22 and end rings 28, 30 are each formed from an elongate member, such as a strip of material, bent into an annular configuration with respective line welds 23, 25, 27 fixing the rings 22, 28, 30 in their annular configuration, as will be discussed in greater detail below. The hub strips 18, 20, hub ring 22, and end rings 28, 30 may be formed from steel, aluminum, or galvanized steel, among other materials. In alternative approaches, the hub ring 22 and the end rings 28, 30 may be fixed in an annular configuration without line welds 23, 25, 27, such as by features integral to the rings, spot welding, parent metal fastening, or mechanical fasteners.

In one approach, each fan blade 26 has a pair of tabs 32, 34 extending through respective slots 36, 38 in the end rings 28, 30. The tabs 32, 34 are bent downward against the end rings 28, 30 to fix the blades 26 to the end rings 28, 30. In alternative embodiments, the fan assembly 10 may be a single-inlet type fan assembly with a hub ring 22 and only one end ring 28. Alternative embodiments may also include attachment points such as rivets or welds that connect the blades 26 to the hub ring 22 and end ring(s) 28 (and 30) without the use of tabs or slots. For example, a Fergas peening process may be used to connect the blades 26 to the end ring(s) 28 (and 30).

FIG. 2 illustrates a method 41 of manufacturing the fan assembly 10 while producing a minimal amount of scrap material. The method 41 includes producing the hub assembly 16 and the end rings 28, 30 at step 43. For some embodiments of the fan assembly 10, such as a single-inlet blower wheel, only one end ring will be produced. The plurality of blades 26 are manufactured at step 45 with a profile that allows the fan assembly 10 to flow air more efficiently, as will be discussed in greater detail below. The hub assembly 16, end rings 28, 30, and blades 26 are then assembled at step 47 to produce the fan assembly 10.

Turning to further details of the fan assembly 10, FIG. 3 illustrates the fan assembly 10 with all but two blades 26A, 26B removed to show the positioning of the hub assembly 16 between the end rings 28, 30. One of the blades illustrated, blade 26A, extends between the end rings 28, 30 with tabs 32A, 34A extending through respective slots 36A, 38A. Blade 26B is connected to the hub ring 22 and the end rings 28, 30 by advancing a tab 34B through slot 24B in the hub ring 22 until the tab 34B passes through slot 38B in the end ring 30. A tab 32B of the blade 26B is then advanced through a slot 36B in the end ring 28 before the tabs 32B, 34B are bent downward against the end rings 28, 30 to fix the blade 26B to the end rings 28, 30. To fix the blade 26B to the hub ring 22, an embossing process forms ridges (not shown) in the blade 26B along paths 42, 44 on either side of hub ring 22 to restrict the hub ring 22 from sliding along the blade

26B. A similar procedure may be performed to install the remaining blades 26 of the fan assembly 10.

In an alternative approach, each slot 24 of the hub ring 22 extends all the way to an outer edge 46 of the hub ring 22. With this configuration, the blade 26B may be connected to the hub assembly 16 and the end rings 28, 30 by passing a center portion 48 of the blade 26B through the slot 24B in a radially inward direction toward the axis 14. The blade 26B is then manipulated to advance tabs 32B, 34B through respective slots 36B, 38B before the blade 26B is secured to the hub ring 22 and the end rings 28, 30 as described above.

As shown in FIG. 3, the hub strips 18, 20 include attachment portions 56A, 56B and 58A, 58B, respectively. With reference to the attachment portion 56A, the attachment portion 56A includes tabs 60, 62 that are welded to the hub ring 22. The tabs of the hub assembly 16 are sized to fit between the slots 24. Although the attachment portions 56A, 56B, 58A, 58B are illustrated with tabs for welding to the hub ring 22, the attachment portions 56A, 56B, 58A, 58B may connect to the hub ring 22 using any number of approaches, including but not limited to nuts and bolts, rivets, spot welds, line welds, and parent metal fastening.

Turning to FIG. 4, an exploded view of the hub assembly 16 is illustrated. The hub 12 has a through bore 80 aligned with the axis of rotation 14 for receiving a motor drive shaft (not shown) and an opening 82 for receiving a set screw (not shown) which secures the hub 12 to the motor drive shaft. The hub 12 may be cold headed from wire, machined from 12L14 steel bar stock, or any other acceptable manufacturing process and/or material. The hub 12 is generally received within openings 84, 86 of the hub strips 18, 20 and is connected to the hub strips 18, 20 using, for example, an orbital press, swedging, and/or staking.

The hub strip 18 includes a body portion 88 extending between the attachment portions 56A, 56B. In the illustrated embodiment, the hub 12 is a separate component from the hub strips 18, 20 and the hub strips 18, 20 have features for aligning the hub strips 18, 20 and receiving the hub 12. More specifically, the body portion 88 includes a hub mounting portion 90 having a concave seat 92 that tapers downward toward a circular flat 94 extending about the opening 84. Similarly, the hub strip 20 has a body portion 96 extending between attachment portions 58A, 58B. The body portion 96 has a hub mounting portion 98 with a concave seat 100 and a circular flat 102 that are complimentary to the seat 92 and the flat 94 of the hub strip 18 such that the hub mounting portions 90, 98 may nest together when the hub assembly 16 is assembled. Alternatively, the hub mounting portions 90, 98 may be flat and lack seats 92, 100 or other structures that nest together.

The hub ring 22 defines a central opening 110 having a center point 112 aligned with the axis 14, as shown in FIG. 4. The hub ring 22 has an outer radius 114 and an inner radius 116 that define a width 113 of the hub ring 22. When the hub strips 18, 20 are connected to the hub ring 22, the body portions 88, 96 extend across the central opening 110 and provide a rigid connection between the hub 12 and the hub ring 22.

As shown in FIG. 5, the blade 26 has a body portion 130 with a pair of opposed end portions 132, 134. The blade 26 has an outlet portion 140 with an outlet angle and an inlet portion 142 with an inlet angle. Preferably, the outlet angle is different than the inlet angle, although the outlet portion 140 and the inlet portion 142 may have similar angles. In the illustrated embodiment, the blade 26 has a compound radius design, with the outlet portion 140 and inlet portion 142 each having a different radius of curvature, although other

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embodiments may have a similar radius of curvature for portions **140**, **142**. The blade **26** moves more air than traditional fan blade designs for a given blade size, which allows a fan assembly utilizing the blade **26** to have fewer blades while flowing the same amount of air as traditional fan assemblies. Conversely, a fan assembly utilizing the blade **26** and having the same number of blades as a traditional fan assembly will flow a greater amount of air and generate a higher static pressure than a traditional fan assembly.

The blade **26** may be formed using, for example, roll forming or stamping. To control consistency during mass production, the blade **26** may be embossed after forming to limit spring-back of the blade **26** and provide more consistent tolerances of the blade **26**. Further, the blade **26** may be made from a number of materials, including but not limited to galvanized steel, aluminum, and plastic. For plastic blades, a rigid or semi-rigid plastic may be chosen, such as polypropylene. A plastic blade may be molded or extruded.

As shown in FIG. 6, the blade **26** extends between an outlet tip **150** and an inlet tip **152** that travel along an outer diameter **154** and inner diameter **156**, respectively, as the fan assembly **10** rotates. In one approach, the outer diameter **154** is in the range of approximately 9.66 inches to approximately 11.80 inches, preferably 10.73 inches. A ratio of the outer diameter **154** to the inner diameter **156** is preferably within the following range:

$$1.0 < \frac{\text{Diameter}_{154}}{\text{Diameter}_{156}} \leq 2.0$$

In other approaches, the diameters **154**, **156** may be increased or decreased with corresponding adjustments to the other dimensions of the blade **26** to accommodate different applications of the blade **26**. The outlet portion **140** of the blade **26** extends inward from the outlet tip **150** along a plane **158**. The plane **158** is oriented at an angle **160** relative to the outer diameter **154**, the angle **160** being in the range of approximately 100° to approximately 180° , preferably 160° . The outlet portion **140** converges with a plane **164** that extends parallel to a plane **174**. A blade camber distance **168** separates the plane **164** from the plane **174**, the blade camber distance **168** being in the range of approximately 0.150 inches to approximately 0.375 inches, preferably 0.287 inches. The plane **174** extends a chord distance **176** between the tips **150**, **152**. The chord distance **176** is in the range of approximately 0.890 inches to approximately 1.088 inches, preferably approximately 0.989 inches. A ratio of the chord distance **176** to the blade camber distance **168** is preferably within the following range:

$$1.5 \leq \frac{\text{Distance}_{176}}{\text{Distance}_{168}} \leq 10.0$$

Given this chord/camber ratio and the chord distance **176**, the blade camber distance **168** for a desired chord/camber ratio may be calculated by dividing the chord distance **176** by the desired chord/camber ratio. A plane **180** generally extends along a radius of the diameters **154**, **156**, and intersects blade tip **150**. The plane **174** is oriented at a blade setting angle **166** relative to the plane **180**. The blade setting angle **166** is in the range of approximately 10° to approximately 40° , preferably approximately 27.4° . An inlet portion **142** extends away from the plane **164** and converges with a

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plane **170** at the inlet tip **152**. The plane **170** is oriented at an angle **172** relative to the inner diameter **156**, the angle **172** being in the range of approximately 45° to approximately 70° , preferably 63° . Given the outer diameter **154**, the ratio of the outer diameter **154** to the inner diameter **156**, and the blade setting angle **166**, the chord distance **176** can be determined using the following equation:

$$\text{Distance}_{176} = \frac{\text{Diameter}_{154}}{2} \cos(\text{Angle}_{166}) \pm \sqrt{\left(\frac{\text{Diameter}_{156}}{2}\right)^2 - \left(\frac{\text{Diameter}_{154}}{2}\right)^2 \sin^2(\text{Angle}_{166})}$$

Another embodiment of a fan assembly **300** having a hub assembly **302** is shown in FIG. 7. The fan assembly **300** is similar to the fan assembly **10**, with the exception that the hub strips **304**, **306** have attachment portions **308A**, **308B** and **310A**, **310B** which lack tabs for engaging a hub ring **312**. Instead, the attachment portions **308A**, **308B** and **310A**, **310B** have slots **313**, **314** which align with slots **316** of the hub ring **312**. To secure the hub strips **304**, **306** to the hub ring **312**, each blade **318** is passed through one of the plurality of slots **313** (or **314**) and one of the plurality of slots **316** before ridges are embossed in the blade **318** to fix the hub strip **304** (or **306**) and the hub ring **312** between the ridges, as discussed above with respect to fan assembly **10**. The hub strips **304**, **306** may alternatively be connected to the hub ring **312** using welds, rivets, or other approaches. Further, slots **313** (or **314**) and the slots **316** may extend to an outer edge **319** of the hub ring **316** to accommodate radial insertion of the blades **318**, as discussed above.

FIG. 8 illustrates a method **400** of producing a product, such as the end rings **28**, **30** or the hub assemblies **16**, **302** of the fan assemblies **10**, **300**, while generating a minimal amount of scrap material. The method **400** comprises a ring forming sub-method **402**, explained with reference to FIGS. **9-13B**, and a finishing sub-method **404**, explained with reference to FIGS. **1** and **14-16**. Although the method **400** is described in steps, it will be appreciated that the steps may be modified, combined, removed, or performed in a different order than the order presented. Further, additional or fewer actions may be performed at each step without departing from the teachings of this disclosure. If the method **400** is used to produce an orifice, end ring, or hub assembly, the method **400** provides a material yield of nearly 100%, which is at least a 33% improvement over traditional processes.

In one approach, the method **400** utilizes a ring forming device **500**, shown in FIG. **9**, to shape a raw material into a ring during the ring forming sub-method **402**. For example, the raw material may be a strip of material **540** (see FIG. **10**) such as aluminum, steel, galvanized steel, coated steel, or other materials which can be advanced from a coil of the raw material. In the illustrated embodiment, the ring forming device **500** is connected to a roll forming machine **502** to utilize rotation of a shaft **504** of the roll forming machine **502**. In another embodiment (not shown), the ring forming device **500** may be a stand-alone device with a dedicated drive motor. Returning to FIG. **9**, the strip of material **540** is advanced between feed rollers **506**, **508**, **510** and into slots **512**, **514** on a pair of forming rollers **516**, **518**. The feed rollers **506**, **508** and forming rollers **516**, **518** are driven to advance the strip of material **540** through the ring forming device **500**. More specifically, a powered roller drives the feed rollers **506**, **508**. Similarly, a drive shaft **504** rotates a drive gear **520** and the roller **516** which is attached to the

drive gear 520. Rotation of the drive gear 520 rotates follower gear 522 and the roller 518 connected thereto in an opposite direction. A pair of arms 524, 526 hold the rollers 516, 518 relative to one another, with plates 528, 530 rigidly fixing the arms 524, 526 together.

The ring forming sub-method 402 begins at step 406 where a raw material and dimensions of the raw material are chosen. In the illustrated approach, the raw material is the strip of material 540 having a generally flat cross-section with a width 542 and a thickness 544 (see FIG. 10). At step 408 in FIG. 8, the strip of material 540 is advanced into the ring forming device 500, as shown in FIG. 10. The strip of material 540 has a center line 546 extending between central axes 548, 550 of the forming rollers 516, 518 as the strip of material 540 is fed through the rollers 506, 508, 510. To adjust the radius of the ring produced from the strip of material 540, the ring forming device 500 includes a radius adjustment device 552 that adjusts the position of the arms 524, 526 above a base 554 of the device 500. The radius adjustment device 552 comprises a threaded bolt (not shown) and a nut 551 engaged with threads of the bolt. The plate 528 rests upon the threaded bolt such that rotation of the nut 551 advances/retracts the bolt relative to the base 554 and elevates/lowers the arms 524, 526 a distance 556 above the base 554.

Adjusting the distance 556 between the arms 524, 526 and the base 554 rotates the arms 524, 526 and the roller 518 about the central axis 548 of the roller 516. By adjusting the position of the arms 524, 526, the angle at which the roller 518 bends the strip of material 540 can be adjusted. More specifically, rotating the arms 524, 526 counterclockwise about the axis 548 decreases the radius of the ring produced from ring forming device 500. Conversely, rotating the arms 524, 526 clockwise about the axis 548 increases the radius of the ring.

With reference to FIG. 10, rotating the arms 524, 526 tends to bring the central axis 550 of the roller 518 closer to the centerline 546 of the strip of material 540. Stated differently, rotating the arms 524, 526 counterclockwise about the axis 548 moves the roller 518, decreases distance 558, and increases distances 566, 568. This causes the strip of material 540 to strike the roller 518 closer to the equator of the roller 518 and bend at a relatively sharp angle away from the roller 518. Conversely, rotating the arms 524, 526 clockwise increases the distance 558 and decreases the distances 566, 568. This causes the strip of material 540 to strike the roller 518 farther from the equator of the roller 518 and bend at a relatively softer angle away from the roller 518.

In the illustrated embodiment, the rollers 506, 508, 510, 516 of the ring forming device 500 do not change position as the radius adjustment device 552 is used to adjust the position of the arms 524, 526. Given the stationary nature of the rollers 510, 516, rotating the arms 524, 526 does not change the distances 560, 563, 564. In alternative embodiments of the ring forming device 500, the positions of one or more of the rollers 506, 508, 510, 516 may be adjustable in combination with, or in place of, movement of the roller 518.

In addition to the radius adjustment device 552, slot depths 570, 572, 574, 576, 578 of the feed rollers 506, 508, 510, 516, 518 also control the radius of the ring produced from the strip of material 540, as shown in FIG. 10. The slot depths 570, 572, 574, 576, 578 are selected to position a predetermined amount of the width 542 of the strip of material 540 within the slots 512, 514 of the rollers 516, 518. Further, the slots of the rollers 506, 508, 510, 516, 518 are

sized to accommodate the thickness 544 of the strip of material 540. In one approach, the width 542 of the strip of material 540 received within the slot 514 as the material 540 passes through the roller 518 is in the range of between approximately 20% and 90% of the total width 542. This engagement between the roller 518 and the strip of material 540 tends to limit flexing of the strip of material 540 as the strip of material 540 advances through the ring forming device 500.

At step 410 in FIG. 8, the strip of material 540 is advanced toward the slotted rollers 516, 518 until a leading end portion 590 contacts the roller 518 and curls upward, as shown in FIG. 11. The leading end portion 590 may include a rounded nose 592 and a flat 594 for contacting the roller 518 and directing the leading end portion 590 upward. In this manner, the roller 518 acts as a curling shoe to curl the strip of material 540. Curling the strip of material 540 compresses a radially inner portion 596 of the strip of material 540 while tensioning a radially outer portion 598 located across the center line 546.

During step 410 in FIG. 8, the strip of material 540 continues to be advanced into the ring forming device 500 until the leading end portion 590 makes a complete loop and the strip of material 540 forms a ring 610, as shown in FIG. 12. The ring 610 can extend in a generally annular configuration for greater or less than 360° as desired for a particular application. The ring 610 has inner and outer radii 614, 616 and a ring width 613 defined between the inner and outer radii 614, 616. The process of bending the strip of material 540 into the ring 610 may create a wave in the ring 610 along the ring width 613. To minimize the waviness of the ring 610, the dimensions of the strip of material 540 may be selected to provide a predetermined ratio of the inner radius 614 to the width 542 (see FIG. 10). For example, the ratio of the inner radius 614 to the width 542 may be in the range of approximately 0.5:1 to approximately 46:1. Further, the dimensions of the strip of material 540 may be selected to provide a predetermined ratio of the outer radius 616 to the width 542. For example, the ratio of the outer radius 616 to the width 542 may be in the range of approximately 2.5:1 to approximately 48:1. These ratios can be adjusted to accommodate different thicknesses 544 of the strip of material 540, as well as different materials, chemistries, and material treatments. The following table presents exemplary ratios for several products having a ten-inch outer diameter:

Type of Product	Ratio of Ring Inner Radius to Strip Width	Ratio of Ring Outer Radius to Strip Width
Forward Curved Wheel	18:1	20:1
Forward Curved Strip Wheel	38:1	40:1
Backward Inclined Wheel	0.5:1	2.5:1
Backward Curved Wheel	0.5:1	2.5:1
Mixed Flow Wheel	8:1	10:1
Orifices and Inlet Rings	3:1	5:1

Continued advancing of the strip of material 540 into the ring forming device 500 produces a helix 620 at step 410, as shown in FIG. 13. The helix 620 wraps around a center axis 612 and is directed to the side of the ring forming device 500. At step 412 in FIG. 8, the ring 610 is cut from the helix 620 and at step 414, mating ends of the ring 610 are formed. Steps 412 and 414 may be combined such that cutting the ring 610 from the helix 620 forms one or both of the mating

ends of the ring 610. For example, as indicated in FIG. 13, the ring 610 may be cut from the helix 620 by creating bevel cuts along paths 622, 624 so that ends 626, 628 of the ring 610 can be flush with one another after the ring 610 is removed from the helix 620. Other approaches may be used to remove the ring 610 from the helix 620, such as using a radial cut. In one approach, the strip of material 540 continues to be advanced into the ring forming device 500 to generate a larger helix 620 with several rings 610 before the helix 620 is separated from the strip of material 540. The rings 610 may then be cut from the helix 620.

At step 416 in FIG. 8, the mating ends 626, 628 are optionally joined together using, for example, a YAG laser-welding procedure. As shown in FIG. 13A, the ring 610 has mating ends 626, 628 joined together at a weld 630. Joining the mating ends 626, 628 of the ring holds the ring 610 in an annular configuration and provides a rigid structure for subsequent processes, such as for connecting the fan blades 26 to the ring 610. For some applications of the ring 610, such as an orifice for a blower housing assembly, the mating ends 626, 628 need not be joined together.

The mating ends of the ring may be joined together using alternative approaches, such as spot welding, parent metal fastening, or mechanical fasteners. For an approach such as spot welding, the ring 610 may be longer than 360° so that there are overlapping portions of the ring 610 that can be spot welded together. For example, the ring 610 may have end portions 632, 634 that overlap and are joined at a spot weld 636, as shown in FIG. 13B. If the ring 610 with overlapping portions 632, 634 is used in a fan assembly application, such as for the hub ring 22 of the fan assembly 10 (see FIG. 1), it is preferred to utilize at least one other ring 610 with overlapping portions in the fan assembly 10, such as the end rings 28 and 30, to evenly balance the fan assembly 10. More specifically, the overlapping portions 632, 634 of the rings 610 are spaced evenly around the fan assembly 10, i.e., the overlapping portions of the hub ring 22 and end rings 28, 30 would each be separated by approximately 120° from one another around the circular profile of the fan assembly 10. Weights or other corrective measures could be then applied to the fan assembly 10 to balance the fan assembly 10 after the blades 26 have been installed.

At this point, the completed ring 610 enters the finishing sub-method 404 of FIG. 8, which will be described in greater detail with respect to FIGS. 1 and 14-16. If the final product is to be a venturi or orifice at step 418 in FIG. 8, the shape of the orifice may be formed at step 420. The orifice can be utilized on mixed flow fans, blower housing inlets, backward inclined centrifugal fan assemblies, and backward curved centrifugal fan assemblies, to name a few applications. In one approach the ring 610 can be formed into an orifice 700, shown in FIGS. 14 and 15, at step 420 in FIG. 8. The ring 610 can be formed into an orifice 700 using spinning or forming in a die. The orifice 700 includes ends 702, 704 that correspond to the ends 626, 628 of the ring 610 in FIG. 13. The ends 702, 704 are joined at a weld 706, which was applied at step 416 in FIG. 8 before the ring 610 entered the finishing sub-method 404. The orifice 700 includes a neck portion 708 defining an inlet 710 and a flange portion 712 for connecting to the associated fan assembly, blower housing structure, or other component. In another approach, the ring forming device 500 simultaneously bends the strip of material 540 into the ring 610 and forms features of an orifice into the ring 610 such that a substantially complete orifice 700 exits the ring forming device 500. In this approach, the rollers 506, 508, 510, 516, 518 are configured to impart the desired curvature of the

orifice 700, such as the neck portion 708, as the strip of material 540 is advanced through the ring forming device 500. The orifice 700 may be made from galvanized steel, aluminized steel, aluminum, or stainless steel, among other materials.

If the ring 610 is to become an end ring at step 422 in FIG. 8, the ring 610 may be shaped at step 424. For example, the end ring 28 of the fan assembly 10 in FIG. 1 may not require additional forming. By contrast, a backward inclined centrifugal fan assembly 800, as shown in FIG. 16, has an end ring 802 with an integral orifice 804 formed using a die press or other forming method at step 424. The forming of the orifice 804 into the end ring 802 is one example of the shaping that may occur at step 424 in FIG. 8. Like the orifice 700, the end ring 802 has ends 806, 808 that correspond to the ends 626, 628 of the ring 610. The ends 806, 808 are joined at a weld 810, or other joining method, which was applied at step 416 in FIG. 8 before the ring 610 entered the finishing sub-method 404.

At step 426 in FIG. 8, attachment points for connecting fan blades to the end ring are added to the end ring. For example, slots may be formed in the end ring if the end ring is similar to the end ring 28 of the fan assembly 10. For the end ring 802 of the fan assembly 800, rivets 812 are used to connect a plurality of blades 814 to the end ring 802.

If the ring 610 is to become part of a hub assembly at step 428 in FIG. 8, the ring 610 is used as a hub ring and shaped at step 430 if needed. Further, fan blade attachment points may be formed on the hub ring at step 430. A hub assembly produced at step 428 using the ring 610 may replace a back disc for, among other applications, a forward curved single inlet centrifugal fan assembly, a backward inclined centrifugal fan assembly, a backward curved centrifugal fan assembly, and a mixed flow fan assembly. A centrifugal fan assembly produced using the ring 610 may be any type of centrifugal air moving device having forward curved blades or other blade configurations. Similarly, a hub assembly produced using the ring 610 can replace a central disc for, among other applications, a forward curved double inlet centrifugal fan assembly or an end disc for a forward curved tangential fan assembly.

For example, the ring 610 may be used as a hub ring 22 of the fan assembly 10 with a minimal amount of shaping and the addition of slots 24 at step 430. Similarly, the ring 610 may be used as a hub ring 824 of the fan assembly 800 with minimal shaping and the addition of holes to receive rivets 813 which secure the plurality of blades 814 to the hub ring 824. As shown in FIG. 16, ends 826, 828 of the hub ring 824 correspond to ends 626, 628 of the end ring 610.

At step 432 in FIG. 8, one or more hub strips for joining to the hub ring may be stamped from a coil of material, such as hub strips 18, 20 of the fan assembly 10 or the hub strips 820, 822 of the fan assembly 800. At step 433, the one or more hub strips may be shaped, such as shaping the hub mounting portions 90, 98 of the hub strips 18, 20 of the fan assembly 10 (see FIG. 4). By contrast, the hub strips 820, 822 are illustrated in FIG. 16 without mating portions, such that shaping the hub strips 820, 822 at step 433 is unnecessary.

At step 434 in FIG. 8, a hub may be connected to the hub strips, such as connecting a hub 823 of the fan assembly 800 to the hub strips 820, 822. For rigidity purposes and/or to influence the resonance frequency of a fan assembly, the hub strips 820, 822 may be joined together before the hub strips 820, 822 and the hub 823 are connected to the hub ring 824. The hub strips 820, 822 may be joined together using welds, rivets, or other approaches. In some applications of the

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method 400, the step 434 is not performed, such as when the hub 823 is integrally formed with one of the hub strips 820, 822. Further, the step 434 may be omitted when a fan assembly does not utilize a hub, such as when the hub strips 820, 822 have a mounting point or feature for direct attachment to a shaft or clamp.

At step 436 in FIG. 8, the attachment portions of the hub strips are connected to the hub ring. For example, the tabs 60, 62 of the hub strip 18 (see FIG. 3) are spot welded or joined by some other method to the hub ring 22. The hub strip 820, by contrast, has attachment portions 830, 832 connected to the hub ring 824 using rivets 834.

Returning to FIG. 8, if the ring 610 is to be used for a product other than an orifice, an end ring, or a hub assembly, the ring 610 goes on to subsequent processing at step 438 to produce the desired product.

It will be understood that various changes in the details, materials, and arrangements of parts and components which have been herein described and illustrated in order to explain the nature of the fan assembly and method may be made by those skilled in the art within the principle and scope of the fan assembly and method as expressed in the appended claims. Furthermore, while various features have been described with regard to a particular embodiment or a particular approach, it will be appreciated that features described for one embodiment also may be incorporated with the other described embodiments.

What is claimed is:

1. A fan assembly comprising:
 - a plurality of elongated metallic fan blades, each blade being curved and including:
 - a first end portion;
 - a second end portion opposite the first end portion;
 - a length extending between the first and second end portions;
 - an inlet portion of the blade having a first radius of curvature;
 - an outlet portion of the blade having a second radius of curvature different than the first curvature;
 - a straight inlet edge of the inlet portion extending from the first end portion to the second end portion; and
 - a straight outlet edge of the outlet portion extending from the first end portion to the second end portion;
 - a first metallic end ring connected to each of the fan blades at the first end portion thereof, the first metallic end ring having a first radially inner edge and a first radially outer edge;
 - a second metallic end ring connected to each of the fan blades at the second end portion thereof, the second metallic end ring having a second radially inner edge and a second radially outer edge;
 - wherein the straight inlet edges of the blades are radially inward from the first and second radially inner edges of the first and second metallic end rings so that the inlet portions of the blades extend radially inward from the first and second metallic end rings; and
 - a metallic hub support connected to the fan blades intermediate the first and second end rings and spaced from the end rings along the blades.
2. The fan assembly of claim 1 wherein the hub support includes a plurality of through openings spaced about a circular outer periphery of the hub support and the fan blades extend through the openings of the hub support.
 3. The fan assembly of claim 2 wherein each through opening includes a curved slot, the curved slot having a radially inner closed end and a radially outer closed end and

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the curved slot extends between the radially inner closed end and the radially outer closed end.

4. The fan assembly of claim 1 wherein each fan blade includes an intermediate portion connecting the inlet portion and the outlet portion, the intermediate portion having a third radius of curvature different than the first radius of curvature and the second radius of curvature.

5. The fan assembly of claim 1 wherein each fan blade has a cross-section perpendicular to the length and the fan blade includes a concave leading surface and a convex trailing surface and a uniform thickness between the leading surface and the trailing surface throughout the entire cross section of the fan blade.

6. The fan assembly of claim 1 wherein each fan blade includes an inlet edge, an outlet edge, a concave leading surface extending from the inlet edge to the outlet edge, and a convex trailing surface extending from the inlet edge to the outlet edge.

7. The fan assembly of claim 1 wherein the first and second end rings are flat.

8. The fan assembly of claim 1 wherein the first and second metallic end rings each include a pair of ends joined together with a weld.

9. The fan assembly of claim 1 wherein the fan blades are forward inclined fan blades each having an inlet tip and an outlet tip, the outlet tip being forward of the inlet tip as the fan turns.

10. The fan assembly of claim 1 wherein the hub support includes a hub configured to be connected to a drive shaft of a motor.

11. The fan assembly of claim 1 wherein the first and second end rings include slots sized to receive the first and second end portions of the fan blades and the first and second end portions of the fan blades include tabs bent against the first and second end rings.

12. The fan assembly of claim 1 wherein the hub support includes a hub ring connected to the fan blades and a hub strip having an attachment portion connected to the hub ring and a body portion extending radially inward from the attachment portion.

13. A fan assembly comprising:
 - a plurality of elongated metallic fan blades, each blade being curved and including:
 - a first end portion;
 - a second end portion opposite the first end portion;
 - a length extending between the first and second end portions;
 - an inlet portion of the blade having a first radius of curvature; and
 - an outlet portion of the blade having a second radius of curvature different than the first curvature;
 - a first metallic end ring connected to each of the fan blades at the first end portion thereof;
 - a second metallic end ring connected to each of the fan blades at the second end portion thereof;
 - a metallic hub support connected to the fan blades intermediate the first and second end rings and spaced from the end rings along the blades;
- wherein the hub support includes a hub ring connected to the fan blades and a hub strip having an attachment portion connected to the hub ring and a body portion extending radially inward from the attachment portion; and
- wherein the hub strip has an elongate configuration with the attachment portion at one end of the hub strip and another attachment portion connected to the hub ring at an opposite end of the hub strip.

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14. The fan assembly of claim 13 wherein the hub ring includes a plurality of through openings spaced about a circular outer periphery of the hub ring and the fan blades extend through the openings of the hub ring.

15. The fan assembly of claim 14 wherein each through opening includes a curved slot, the curved slot having a radially inner closed end and a radially outer closed end and the curved slot extends between the radially inner closed end and the radially outer closed end.

16. The fan assembly of claim 13 wherein each fan blade includes an intermediate portion connecting the inlet portion and the outlet portion, the intermediate portion having a third radius of curvature different than the first radius of curvature and the second radius of curvature.

17. The fan assembly of claim 13 wherein each fan blade has a cross-section perpendicular to the length and the fan blade includes a concave leading surface and a convex trailing surface and a uniform thickness between the leading surface and the trailing surface throughout the entire cross section of the fan blade.

18. The fan assembly of claim 13 wherein each fan blade includes an inlet edge, an outlet edge, a concave leading

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surface extending from the inlet edge to the outlet edge, and a convex trailing surface extending from the inlet edge to the outlet edge.

19. The fan assembly of claim 13 wherein the first and second end rings are flat.

20. The fan assembly of claim 13 wherein the first and second metallic end rings each include a pair of ends joined together with a weld.

21. The fan assembly of claim 13 wherein the fan blades are forward inclined fan blades each having an inlet tip and an outlet tip, the outlet tip being forward of the inlet tip as the fan turns.

22. The fan assembly of claim 13 wherein the hub support includes a hub configured to be connected to a drive shaft of a motor.

23. The fan assembly of claim 13 wherein the first and second end rings include slots sized to receive the first and second end portions of the fan blades and the first and second end portions of the fan blades include tabs bent against the end rings.

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