

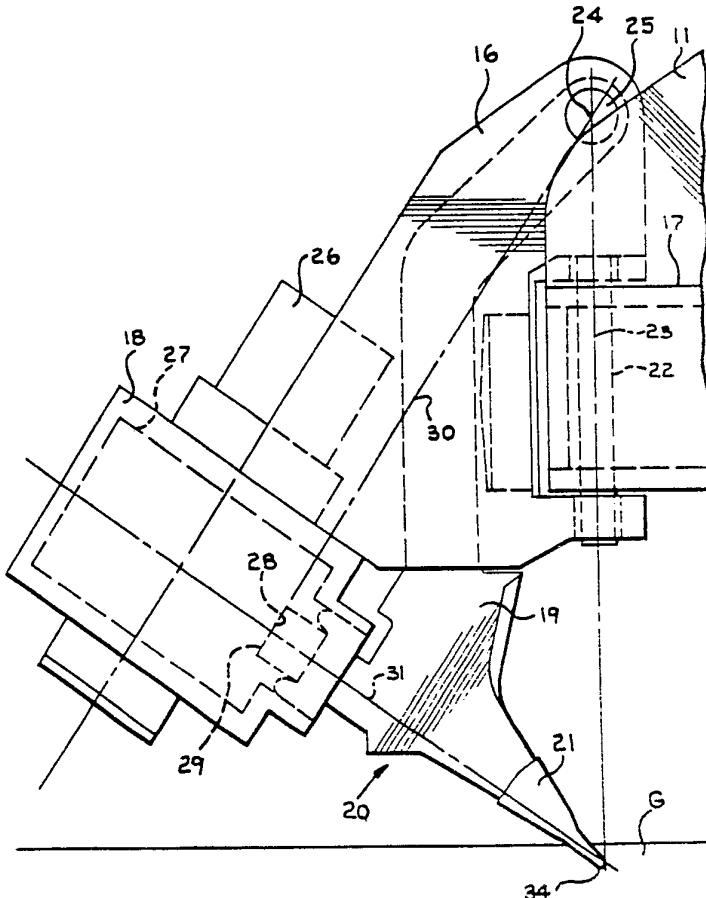


INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

(51) International Patent Classification 3: A01B 35/00	A1	(11) International Publication Number: WO 80/01637 (43) International Publication Date: 21 August 1980 (21.08.80)
(21) International Application Number: PCT/US79/00077 (22) International Filing Date: 12 February 1979 (12.02.79) (71) Applicant (for all designated States except US): CATER-PILLAR TRACTOR CO. [US/US]; 100 Northeast Adams Street, Peoria, IL 61629 (US). (72) Inventors; and (75) Inventors/Applicants (for US only): COBB, Delwin, E. [US/US]; 6401 North Brookwood Lane, Peoria, IL 61614 (US). LIVESAY, Richard, E. [US/US]; 1011 Deerbrook Drive, Peoria, IL 61614 (US). (74) Agents: WALTERS, Ralph, E.; 100 Northeast Adams Street, Peoria, IL 61629 (US) et al.		(81) Designated States: GB, JP, US. Published <i>With international search report</i>

(54) Title: IMPACT RIPPER**(57) Abstract**

An improved ripper assembly (10) where in a ripper shank (19) is mounted to the beam (11) of a tractor by a clevis pin (22) so as to minimize side forces acting on the shank pin (25). The vertical axis of the clevis pin may be aligned with the distal end (34) of the ripper tip (21) and the horizontal axis (24) of the shank pivot pin (25). The impact surfaces (28, 29) of the impacting mechanism (18) and shank (19) preferably define a plane of abutment in which lies a radius (30) from the horizontal shank pivot pin axis (24). This plane is further preferably accurately perpendicular to the ripper tip axis (31) extending perpendicularly to the radius (30). The arrangement provides a free pivoting action of the ripper assembly about the vertical axis of the clevis pin so as to effectively minimize side loads for improved extended life of the ripper assembly (10).



FOR THE PURPOSES OF INFORMATION ONLY

Codes used to identify States party to the PCT on the front pages of pamphlets publishing international applications under the PCT.

AT	Austria	LI	Liechtenstein
AU	Australia	LU	Luxembourg
BR	Brazil	MC	Monaco
CF	Central African Republic	MG	Madagascar
CG	Congo	MW	Malawi
CH	Switzerland	NL	Netherlands
CM	Cameroon	NO	Norway
DE	Germany, Federal Republic of	RO	Romania
DK	Denmark	SE	Sweden
FR	France	SN	Senegal
GA	Gabon	SU	Soviet Union
GB	United Kingdom	TD	Chad
HU	Hungary	TG	Togo
JP	Japan	US	United States of America
KP	Democratic People's Republic of Korea		

-1-

Description

Impact Ripper

Technical Field

This invention relates to earthworking machines
5 and more specifically to impact rippers.

Background Art

In one form of conventional earth ripper, a ripper tip is mounted to a shank which is pivotally mounted to the tractor beam by a shank pivot pin. As a result of 10 side loads acting on the ripper during the ripping operation, the shank pivot pins have worn so that the pins, at times, were lost during the ripping operation, presenting serious safety and maintenance problems.

A number of different pivoting arrangements have 15 been utilized in connection with such ripping devices. Illustratively, in U. S. Letters Patent 2,797,629 of Carlton O. Kelley, a pivoted shank subsoil contour plow is shown wherein the distal tips of the plow blades are disposed directly below and in vertical alignment with the pivot 20 pin mounting the blades to a frame member so as to prevent side draft at the tips of the plows.

Donald J. Larson, in U. S. Letters Patent 2,998,965, which patent is owned by the assignee hereof, shows a push block for a tractor-mounted ripper mounted to a tractor by 25 a pair of draft arms. The ripper shank is provided with a ripper tip and connected with the tractor beam by a clevis pivotally connected to the beam by a clevis pin. The ripper tip is spaced substantially rearwardly of the axis of the clevis pin.

30 In U. S. Letters Patent 3,031,175 of Mack Wooldridge, a ripper mounting for twin crawlers is shown wherein the ripper tip is spaced rearwardly of the vertical axis of the clevis pin.



-2-

Albert G. Bodine shows a sonic rectifier coupling for rock cutting apparatus in U. S. Letters Patent 3,367,716, wherein an impact device is engageable with an impact surface of the ripper shank at an angle there-
5 to.

In U. S. Letters Patent 3,770,322, Delwin E. Cobb et al show an apparatus for fracture of material in situ with stored inertial energy which is cyclically delivered on demand to the work tool, and which energy is stored
10 in a large flywheel and delivered to the tool by suitable transmission means. The single pivot point disclosed by Cobb et al is indicated as preferably located ahead of the cutting tip to obtain the desired motion.

Darrel M. Wooldridge, in U. S. Letters Patent 15 4,044,838, shows an automatic control for a ripper tool which is disposed substantially rearwardly of the majority of the mounting pins of the assembly.

Disclosure of Invention

The present invention is directed to overcoming
20 one or more of the problems as set forth above.

The present invention comprehends an improved ripper structure utilizing an impact energy source mounted at the rear of an earthmoving machine, such as a tractor.

The impact ripper assembly is mounted to the
25 beam of the tractor by a clevis-type mounting.

In the illustrated embodiment, the ripper shank is pivoted from a point located on a line parallel to the impact surface and substantially above the surface.

Further, in the illustrated embodiment, the
30 shank pivot pin axis is disposed substantially vertically above the distal tip of the ripper.

Still further in the illustrated embodiment,
the shank pivot pin axis is disposed substantially in
intersecting relationship to the vertical axis of the
35 clevis pivot pin pivotally mounting the assembly to the



-3-

beam for swinging movement about a vertical axis so as to provide limited side-to-side movement of the ripper in normal operation thereof.

The ripper includes a rock breaker mechanism for providing desirable impact forces to the ripper tip in the earthworking operation with the impact surface of the hammer of the rock breaker mechanism being accurately in parallel abutting relationship to the impact surface of the ripper tip shank at the point of impact in the operation of the apparatus.

In the illustrated embodiment, the abutment of the impact surfaces defines a plane extending perpendicularly transversely to the ripper tip axis and in which lies a radius from the shank pivot pin axis extending 15 perpendicularly to the ripper tip axis.

Thus, the present structure effectively minimizes wear of the shank pivot pin to provide long trouble-free life of the ripper assembly mounting to the tractor beam. The ripper structure of the present invention is 20 extremely simple and economical of construction while yet providing the highly desirable features discussed above.

Brief Description of Drawing

FIGURE 1 is a fragmentary side elevation of an earthworking apparatus including a ripper structure embodying the invention; and

FIGURE 2 is a fragmentary enlarged side elevation of the ripper structure.

Best Mode for Carrying Out the Invention

In the exemplary embodiment of the invention as disclosed in the drawing, a ripper assembly generally designated 10 is shown as mounted to a beam generally designated 11 mounted on the rear portion 12 of a tractor 13. The tractor includes suitable extensible elements 14 and 15, such as hydraulic cylinder devices, for controlling the



-4-

disposition of the beam and, thus, the positioning of the ripper assembly in the operation thereof. The invention is concerned with the mounting of the ripper assembly 10 to the beam 11 and, in the arrangement of 5 the elements of the ripper assembly relative to such mounting, to provide an improved long life, troublefree operation.

More specifically, as seen in Figure 1, the ripper assembly 10 includes a bracket 16 mounted to the 10 beam 11 by a clevis 17. The bracket carries a rock breaker mechanism 18 for providing an impact force to the shank 19 of a ripper generally designated 20 and provided at its lower end with a ripper tip 21. In normal operation, as the tractor is moved forwardly to draw 15 the ripper into the ground G to be ripped, the tip 21 is impacted by the mechanism 18 so as to provide an improved rock breaking ripping operation for facilitated earth working.

Referring now more specifically to Figure 2, 20 the structural arrangement of the ripper assembly may be more clearly seen. As shown, the clevis is provided with a clevis pin 22 which is arranged to define a substantially vertical axis 23 extending downwardly vertically to the distal end 34 of the ripper tip 21. The clevis pin axis extends vertically upwardly to perpendicularly intercept a horizontal axis 24 of a shank pivot pin 25 pivotally mounted to the bracket 16 and carrying the shank 19 for pivotal swinging about the axis 24. Thus, the ripper tip 21 is positionable about the vertical axis 23 of 25 the clevis pivot and, in the illustrated embodiment, is swingable thereabout approximately 30°, i.e., 15° in each direction from the centered disposition. Concurrently, the ripper tip 21 is swingable about the horizontal axis 24, and more specifically, is arranged to provide an impact 30 rippling action as a result of the impact forces generated thereon by the rock breaker mechanism 18 so as to provide 35



a desired ripping force downwardly into the ground G, as shown in Figure 2 in the normal operation of the ripper assembly.

5 The impact force applying means may comprise a conventional rock breaker mechanism which may include a conventional operating motor 26. The motor drives a conventional flywheel-crankshaft ring impactor 27 well known to those skilled in the art and, thus, no further description thereof need be given here. However, the impactor 10 device defines an impact surface 28 which impinges on a reaction impact surface 29 of the shank 19 in the impacting operation so as to apply an impact force to the shank so as to impact the ripper tip 21 into the ground G, as illustrated in Figure 2.

15 As shown in Figure 2, the surfaces 28 and 29 are accurately parallel so as to define a common plane when in abutment with each other, which plane contains a radius 30 from the horizontal shank pivot pin axis 24 extending perpendicular to the axis 31 of the ripper tip 21.

20 Thus, the invention comprehends mounting the ripper assembly 10 to the beam 11 by means of an improved connection including the clevis pin 22 axially vertically aligned with the distal end 34 of the ripper tip 21. The horizontal shank pin axis, within the scope of the invention, 25 is disposed to substantially intersect the vertical clevis pin axis 23.

30 It is further contemplated within the scope of the invention that the impact surfaces be accurately parallel so as to define a common plane when in abutment in the impacting operation, which plane defines a radius 30 of the shank pivot pin axis 24 and which perpendicularly intersects the ripper tip axis 31 which extends perpendicularly to the radius 30.

35 As discussed above, the vertical clevis pivot pin axis 23 is preferably vertically aligned with the distal end 34 of the ripper tip and the horizontal axis 24



-6-

of the shank pivot pin. The invention comprehends that within the concept of substantial alignment thereof, the axis of the clevis pivot pin 22 may be disposed somewhat forwardly of this line in providing the desired force 5 moment to assure turning of the unit into the rocks, etc. of the ground G with effectively minimized side forces tending to wear the pivot pins of the device.

In the illustrated embodiment, the impact surfaces 28 and 29 are disposed at a 35° angle to the horizontal. Such disposition has been found to provide an 10 excellent impacting operation which, when combined with the improved reduction of side forces acting on the shank pin, provides a long, highly efficient, troublefree life of the ripper assembly.

15 Industrial Applicability

As indicated above, the ripper structure of the present invention may be utilized as an impact energy source, such as mounted at the rear of an earthmoving machine. One form of machine with which the ripper structure may be advantageously utilized is a tractor. 20

Further more specifically, the ripper assembly may be mounted to the beam of the tractor as by clevis-type mounting.

The impact ripper may be utilized as a conversion 25 unit wherein a conventional rockbreaker device is converted to an impact rock trencher. The apparatus, when utilized in producing trenches, advantageously produced vertical sidewalls and efficiently operated as an impact rock cruncher.

Other aspects, objects and advantages of this 30 invention can be obtained from a study of the drawings, the disclosure and the appended claims.

Claims

1. In a ripper assembly (10) having a beam (11) adapted to be secured to an earthworking machine, a clevis pivot pin (22) mounted to said beam to define a vertical pivot axis (23), an impact breaker mechanism 5 (18), a shank (19) provided with a shank pivot pin (25), a ripper tip (21) defining a ripper tip axis (31), and a bracket (16) pivotally mounted to the clevis pivot pin (22) and carrying the impact breaker mechanism (18) to apply an impact force to the shank (19) urging the ripper 10 tip (21) about the axis (24) of the shank pivot pin (25), the improvement comprising

means on said bracket (16) for mounting said shank pivot pin (25) to define a horizontal pivot axis (24) substantially intersecting said clevis pivot pin vertical axis (23). 15

2. The ripper assembly of Claim 1 wherein said clevis pivot pin axis (23) substantially intersects the distal end (34) of the ripper tip (21).

3. The ripper assembly of Claim 1 wherein said 20 impact breaker mechanism is swingable about said shank pin axis (24) and said shank (19) defines an impact surface (29) extending perpendicularly to the ripper tip axis (31), said mechanism defining an impact means (18) impacting against said impact surface (29) accurately perpendicularly 25 to said ripper tip axis (31).

4. The ripper assembly of Claim 1 wherein said impact breaker mechanism is swingable about said shank pin axis (24) and said shank (19) defines an impact surface (29) extending perpendicularly to the ripper tip axis (31), 30 said mechanism defining an impact means (18) impacting against said impact surface (29) accurately perpendicularly



-8-

to said ripper tip axis, said impact surface (29) extending parallel to a radius (30) from said shank pivot pin axis (24) extending perpendicularly to said ripper tip axis (31).

5 5. The ripper assembly of Claim 1 wherein said clevis pivot pin axis (23) substantially intersects the distal end (34) of the ripper tip (21), said impact breaker mechanism is swingable about said shank pin axis (24) and said shank (19) defines an impact surface (29) extending perpendicularly to the ripper tip axis (31), said mechanism defining an impact means (18) impacting against said impact surface (29) accurately perpendicularly to said ripper tip axis (31).

15 6. The ripper assembly of Claim 1 wherein said clevis pivot pin axis (23) substantially intersects the distal end (34) of the ripper tip (21), said impact breaker mechanism is swingable about said shank pin axis (24) and said shank (19) defines an impact surface (29) extending perpendicularly to the ripper tip axis (31), said mechanism defining an impact means (18) impacting against said impact surface (29) accurately perpendicularly to said ripper tip axis, said impact surface (29) extending parallel to a radius (30) from said shank pivot pin axis (24) extending perpendicularly to said ripper tip axis 25 (31).

7. In a ripper assembly (10) having a beam (11) adapted to be secured to an earthworking machine, a clevis pivot pin (22), an impact breaker mechanism (18), a shank (19) provided with a shank pivot pin (25), a ripper tip (21) defining a ripper tip axis (31), and a bracket (16) pivotally mounted to the clevis pivot pin (22) and carrying the impact breaker mechanism (18) to apply an impact force to the shank (19) urging the ripper



tip (21) about the axis (24) of the shank pivot pin (25),
the improvement comprising:

5 means on said bracket (16) for mounting said shank
pivot pin (25) to define a horizontal pivot
axis (24) substantially vertically above the
ripper tip (21); and
means for mounting said clevis pivot pin (22) to
said beam (11) to define a vertical pivot axis
10 (23) substantially intersecting the distal end
(34) of said ripper tip (21).

8. The ripper assembly of Claim 7 wherein said
impact surface (29) extends parallel to a radius (30) from
said shank pivot pin axis (24) extending perpendicularly
to said ripper tip axis (31).

15 9. In a ripper assembly (10) having a beam
(11) adapted to be secured to an earthworking machine,
a clevis pivot pin (22) mounted to said beam to define
a vertical pivot axis (23), an impact breaker mechanism
(18), a shank (19) provided with a shank pivot pin (25),
20 a ripper tip (21) defining a ripper tip axis (31), and a
bracket (16) pivotally mounted to the clevis pivot pin
(22) and carrying the impact breaker mechanism (18) to
apply an impact force to the shank (19) urging the ripper
tip (21) about the axis (24) of the shank pivot pin (25),
25 the improvement comprising

means defining planar, confronting impact force trans-
fer surfaces (28,29) on said breaker mechanism
(18) and shank (19) arranged to extend trans-
versely to said ripper tip axis (31), the
30 abutment of said transfer surfaces defining a
plane in which lies a radius (30) from said
shank pivot pin axis (24) extending perpendicu-
larly to said ripper tip axis (31).



-10-

10. The ripper assembly of Claim 9 wherein said breaker mechanism (18) comprises means defining a first (28) of said force transfer surfaces (28,29) defining a plane containing at all times said radius (30).

5 11. The ripper assembly of Claim 9 wherein said breaker mechanism (18) comprises means defining a first (28) of said force transfer surfaces (28,29) defining a plane containing at all times said radius and said shank (19) includes an anvil portion defining the other (29) of 10 said surfaces (28,29), said other (29) of said surfaces defining a plane extending perpendicularly to said ripper tip axis (31) at all times.

12. The ripper assembly of Claim 9 wherein said breaker mechanism (18) comprises means defining a first 15 (28) of said force transfer surfaces (28,29) defining a plane containing at all times said radius and said shank (19) includes an anvil portion defining the other (29) of said surfaces (28,29), said other (29) of said surfaces defining a plane extending perpendicularly to said ripper 20 tip axis (31) at all times, the line between the distal end (34) of the ripper tip (21) and said shank pivot pin axis (24) extending vertically.

13. The ripper assembly of Claim 9 wherein said breaker mechanism (18) comprises means defining a first 25 (28) of said force transfer surfaces (28,29) defining a plane containing at all times said radius and said shank (19) includes an anvil portion defining the other (29) of said surfaces (28,29), said other (29) of said surfaces defining a plane extending perpendicularly to said ripper 30 tip axis (31) at all times, the line between the distal end (34) of the ripper tip (21) and said shank pivot pin axis (24) extending vertically and being substantially coincident with said clevis pivot pin axis (23).

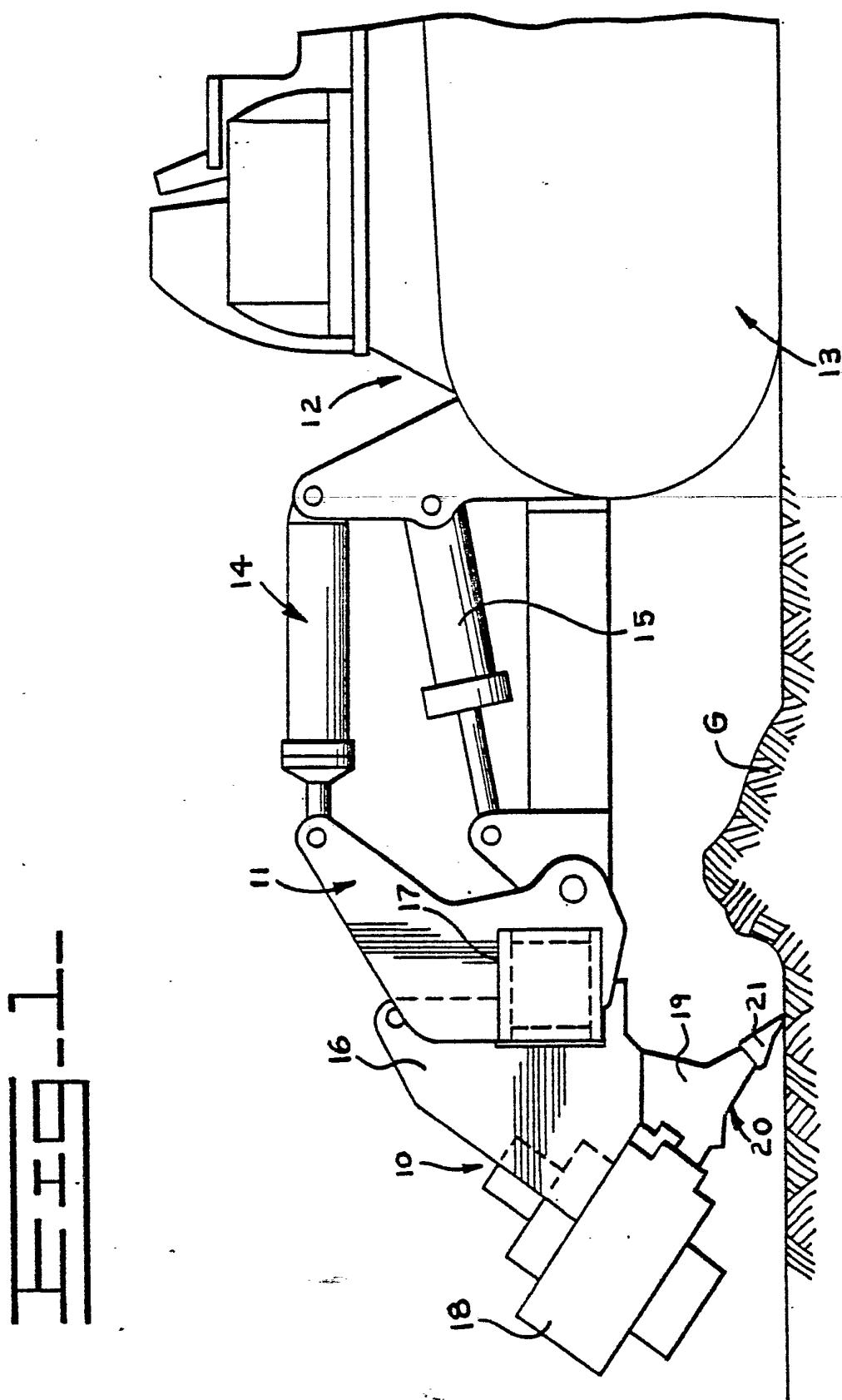


-11-

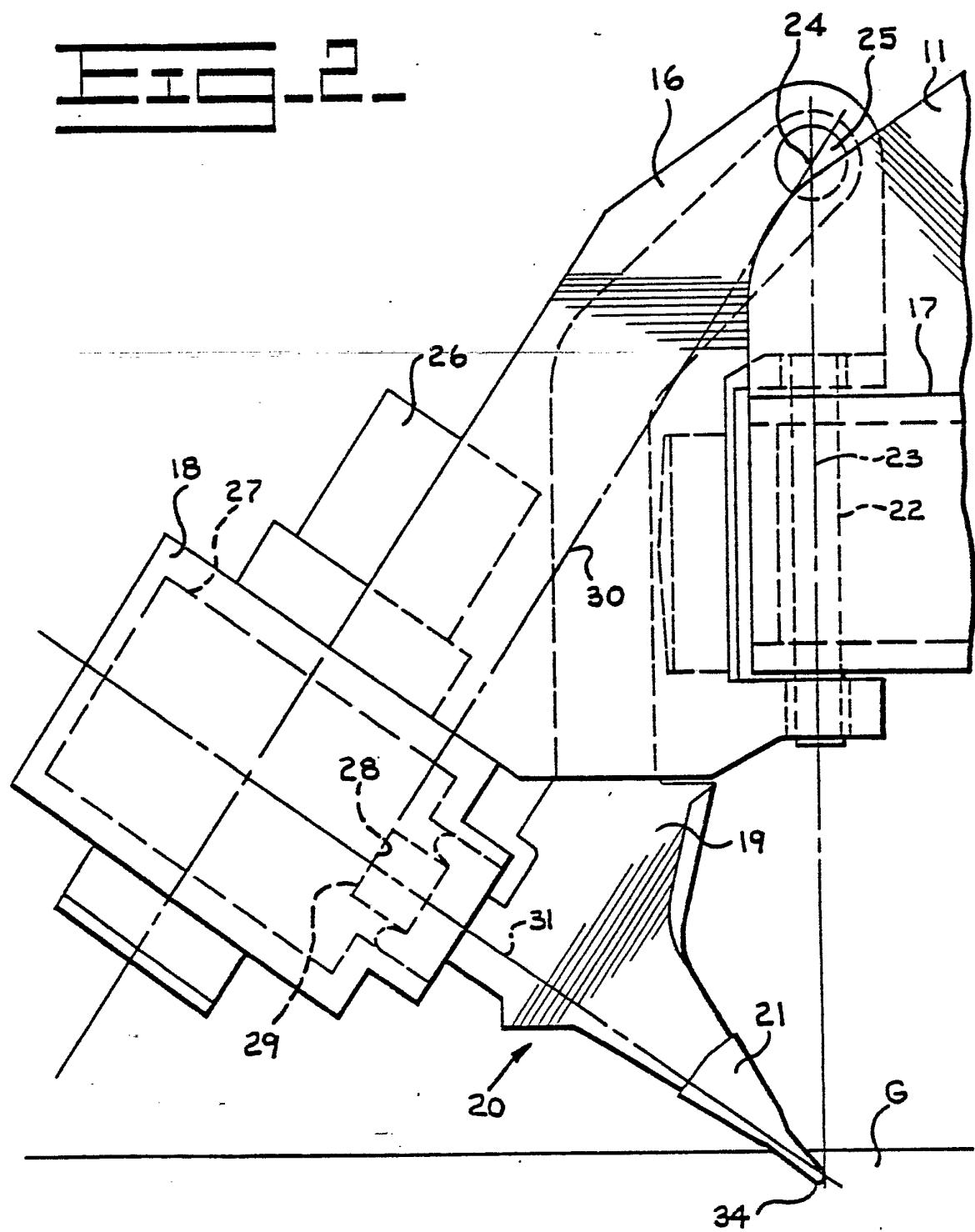
14. The ripper assembly of Claim 9 wherein said breaker mechanism (18) comprises means defining a first (28) of said force transfer surfaces (28,29) defining a plane containing at all times said radius and said shank 5 (19) included an anvil portion defining the other (29) of said surfaces (28,29), said other (29) of said surfaces defining a plane extending perpendicularly to said ripper tip axis (31) at all times, the line between the distal end (34) of the ripper tip (21) and said shank pivot pin 10 axis (24) extending vertically and being substantially coincident with said clevis pivot pin axis (23), said shank (19) and breaker mechanism (18) being swingable as a unit approximately 30° about said clevis pivot pin axis (23).



-1-



-2-



INTERNATIONAL SEARCH REPORT

International Application No. PCT/US79/00077

I. CLASSIFICATION OF SUBJECT MATTER (if several classification symbols apply, indicate all) ³

According to International Patent Classification (IPC) or to both National Classification and IPC

Int. Cl. A01b 35/00
U.S. CL. 299/37,172/40

1600/163)

II. FIELDS SEARCHED

Minimum Documentation Searched ⁴

Classification System	Classification Symbols
U.S.	299/14,36,37 172/40 37/Dig. 18

Documentation Searched other than Minimum Documentation
to the Extent that such Documents are Included in the Fields Searched ⁵

III. DOCUMENTS CONSIDERED TO BE RELEVANT ¹⁴

Category ⁶	Citation of Document, ¹⁶ with indication, where appropriate, of the relevant passages ¹⁷	Relevant to Claim No. ¹⁸
X	US, A, 4,070,064, Published 24 January 1978 Cobb et al	1-14
X	US, A, 4,003,603, Published 18 January 1977 Stemler et al	9-12
A	US, A, 3,698,484, Published 17 October 1972 Kinnan	
A	US, A, 3,770,322, Published 06 November 1973 Cobb et al	
A	US, A, 2,998,965, Published 05 September 1961 Larson	
A	US, A, 3,336,082, Published 15 August 1967 Bodine	
A	SU, A, 568,708, Published 14 November 1977 Lebedev et al	

* Special categories of cited documents: ¹⁵

"A" document defining the general state of the art

"E" earlier document but published on or after the international filing date

"L" document cited for special reason other than those referred to in the other categories

"O" document referring to an oral disclosure, use, exhibition or other means

"P" document published prior to the International filing date but on or after the priority date claimed

"T" later document published on or after the International filing date or priority date and not in conflict with the application, but cited to understand the principle or theory underlying the invention

"X" document of particular relevance

IV. CERTIFICATION

Date of the Actual Completion of the International Search ²

13 November 1979

Date of Mailing of this International Search Report ²

18 DEC 1979

International Searching Authority ¹

ISA/US

Signature of Authorized Officer ²⁰

