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Description

BACKGROUND OF THE INVENTION

[0001] This invention generally relates to key cutting machines. More particularly, the present invention relates to a key cutting machine that has improved alignment, clamping, and cutting mechanisms and to a related method for cutting a notch pattern on a key blank.

[0002] Key duplication systems exist that cut biting patterns in blank keys by tracing the original key to be duplicated or by using a code or extracting the biting patterns from a database. The former type is more prevalent in consumer outlets and the latter is more prevalent in businesses such as automobile dealers and rental agencies.

[0003] But the tracing and code cutting key duplication systems require precise alignment and clamping mechanisms to maintain the ability to accurately cut keys. These issues are rendered more difficult by the use of double-sided keys, which are more common in the automobile industry.

[0004] A key cutting machine described in U.S. Patent No. 5,676,504 utilizes a set of cassettes to address the need for proper alignment and clamping of a key. The cassettes may be selectively inserted into an cassette receiving aperture of the key cutting machine and provide both positional alignment function and clamping function while the key cutting machine cuts a bit notch pattern on a key blank. The key cutting machine described in '504 requires an operator to select different cassettes for different key blanks and different bit notch patterns. The preamble of claim 1 is based on US 5 443 339.

[0005] Thus, there exists a need in the art for improved key duplication systems that can readily align and clamp keys, particularly double-sided keys. There is a related need to reduce the space occupied by such machines and to provide a machine with a small "footprint" that can be easily transported. Such machines must be capable of accurately cutting keys of a diverse nature with a low level of miscuts.

SUMMARY OF THE INVENTION

[0006] The present invention is directed to a key cutting machine according to claim 1. The advantages and purposes of the invention will be set forth in part in the description which follows, and in part will be obvious from the description, or may be learned by practice of the invention. The advantages and purposes of the invention will be realized and attained by the elements and combinations particularly pointed out in the appended claims.

[0007] In accordance with the invention, a key cutting machine comprises a pair of cutting heads and a clamp mechanism. Each cutting head includes a cutting wheel having an edge. An alignment structure is mounted behind each cutting wheel. Each alignment structure has a key guide portion aligned tangent to the edge of a corre-

sponding cutting wheel to engage and position a key blank laterally. The clamp mechanism is positioned between the cutting heads and is configured to hold the key blank stationary.

[0008] The invention is directed to a key cutting machine for cutting a notch pattern on a key blank. The key cutting machine comprises a housing and a pair of cutting heads. The housing includes a top frame and a base. The cutting heads are suspended below the top frame of the housing without being supported on the base of the housing. Each cutting head includes a cutting wheel rotatable to cut the notch pattern on the key blank.

[0009] The invention is directed to a key cutting machine for cutting a notch pattern on a key blank. The key cutting machine comprises a housing, a pair of cutting heads, and a clamp mechanism. The housing includes a top frame. The cutting heads are positioned within the housing. Each cutting head includes a cutting wheel rotatable to cut the notch pattern on the key blank. The clamp mechanism is positioned between the cutting heads. The clamp mechanism includes a lower clamp and an upper clamp. The upper clamp is suspended from the top frame of the housing and is vertically movable relative to the lower clamp to hold the key blank stationary.

[0010] Additional objects and advantages of the invention will be set forth in part in the description which follows, and in part will be obvious from the description, or may be learned by practice of the invention. The objects and advantages of the invention will be realized and attained by means of the elements and combinations particularly pointed out in the appended claims.

[0011] It is to be understood that both the foregoing general description and the following detailed description are exemplary and explanatory only and are not restrictive of the invention, as claimed.

BRIEF DESCRIPTION OF THE DRAWINGS

[0012] The accompanying drawings, which are incorporated in and constitute a part of this specification, illustrate embodiments of the invention and together with the description, serve to explain the principles of the invention. In the drawings,

Fig. 1 is a front view of an embodiment of a key cutting machine according to the present invention;

Fig. 2 is a side view of the key cutting machine of Fig. 1;

Fig. 3 is a front view of a key cutting machine as shown in Fig. 1 illustrating a vertical center line;

Fig. 4 is a section view of a key cutting machine taken along line B-B of Fig. 3 illustrating a clamp mechanism and a tip stop mechanism;

Fig. 5 is a partial perspective view of a key cutting machine according to the present invention illustrating alignment paddles pivotally mounted to cutting heads;

Fig. 6 is an enlarged perspective view of the portion of a key cutting machine enclosed in circle A of Fig. 5; Fig. 7 is a top view of a key cutting machine according to the present invention illustrating a tip stop mechanism;

Fig. 8 is a side view of a cutting machine according to the present invention illustrating a tip stop mechanism;

Fig. 9 is a partial front view of the top of a clamp mechanism illustrating a closed position of the clamp;

Fig. 10 is a partial front view of a clamp mechanism illustrating an open position of the clamp;

Fig. 11 is a perspective view of a key and an anvil;

Fig. 12 is a section view of a key cutting machine taken along line A-A of Fig. 13;

Fig. 13 is a rear view of a key cutting machine according to the present invention;

Fig. 14 is a front view of a key cutting machine according to the present invention illustrating a threaded shaft of a stepper motor for lateral movements;

Fig. 15 is a perspective view of a key cutting machine according to the present invention;

Fig. 16 is a partial perspective view of a key cutting machine according to the present invention where a top frame is removed from Fig. 15;

Fig. 17 is a partial perspective view of a key cutting machine according to the present invention illustrating linear bearings and a top portion of a suspension bracket;

Fig. 18 is a partial perspective view of a key cutting machine according to the present invention illustrating an upper clamp and an anvil;

Fig. 19 is a partial perspective view of the top of a clamp mechanism illustrating a shaft and a cam;

Fig. 20 is a partial perspective view of the top of a clamp mechanism illustrating a block biased by springs;

Fig. 21 is a partial perspective view of another embodiment of a key cutting machine according to the present invention;

Fig. 22 is a partial front view of the key cutting machine of Fig. 21 illustrating pivot arms; and

Fig. 23 is a partial bottom view of the key cutting machine of Fig. 21 illustrating a push block and a spring.

DETAILED DESCRIPTION

[0013] Reference will now be made in detail to the presently preferred embodiment of the present invention, examples of which are illustrated in the accompanying drawings. Wherever possible, the same reference numbers will be used throughout the drawings to refer to the same or like parts. An exemplary embodiment of a key cutting machine of the present invention is designated generally by reference number 20.

[0014] As illustrated in Figs. 1, 2, and 15, key cutting

machine 20 includes a housing having a top frame 24 that is supported by a pair of side frames 26. A pair of suspension blocks 30 are connected to the front of top frame 24 and a pair of suspension blocks 30 are connected to the back of top frame 24. Each suspension block 30 includes an opening 32 to receive and hold a guide rail 34.

[0015] A pair of linear bearings 36, each supported by bearing support 38, are slidably mounted on each guide rail 34 (Figs. 5, 16, and 17). Bearing supports 38 are mounted to a carriage or suspension bracket 40 at the top portion 42 thereof (Figs. 1 and 14-17).

[0016] A pair of cutting heads 28 are suspended from top portion 42 of suspension bracket 40. Each cutting head 28 includes a cutting wheel 29 rotated by a drive motor 31 for cutting a notch pattern on a key blank 33.

[0017] As described above, cutting heads 28 are suspended below linear bearings 36. As a result, the possibility of cutting chips falling on and contaminating linear bearings 36, or any other moving parts of key cutting machine 20, is reduced. In addition, top portion 42 of suspension bracket 40 prevents cutting chips from reaching linear bearings 36 and thereby further reduces the possibility of contamination. Moreover, drive motors 31 preferably rotate cutting wheels 29 in the directions designated by reference number 35 to direct the majority of cutting chips toward a base 23 of key cutting machine 20 and away from linear bearings 36 and other cutting components. Preferably, base 23 includes openings 21 (Figs. 5 and 15-18) for cutting chips to fall through. A tray (not shown) may be positioned below openings 21 to collect cutting chips.

[0018] As mentioned above, both cutting heads 28 are suspended from top portion 42 of suspension bracket 40. In other words, top portion 42 of suspension bracket 40 is common to both cutting heads 28. Accordingly, both cutting heads 28 move longitudinally in unison as suspension bracket 40 moves along guide rails 34 in longitudinal directions designated by reference number 46 (Figs. 2, 4, and 17).

[0019] A stepper motor 48 controls the longitudinal movement of cutting heads 28. As shown in Figs. 4 and 15, stepper motor 48 is suspended from top frame 24. A threaded shaft 50 extends through stepper motor 48 and one end thereof is connected to a block 54 supported on the upper surface of top portion 42 of suspension bracket 40. As illustrated in Fig. 4, a pin 52 connects threaded shaft 50 to block 54. However, other known techniques may be used to connect threaded shaft 50 to block 54. As the motor armature (not shown) of stepper motor 48 rotates in a first direction, threaded shaft 50 moves toward the back of key cutting machine 20 and carries with it suspension bracket 40. Consequently, both cutting heads 28 move longitudinally in unison toward the back of key cutting machine 20. Similarly, both cutting heads 28 move in unison toward the front of key cutting machine 20 as the motor armature of stepper motor 48 rotates in a second direction opposite to the first direction.

[0020] While both cutting heads 28 move longitudinally in unison, their lateral movements are controlled independently. As shown in Figs. 2, 4, 5, 6, and 17, each cutting head 28 has a dove-tail shaped projection 72 protruding from the top surface thereof. Dove-tail shaped projection 72 of each cutting head 28 slidably engages a correspondingly-shaped groove 74 formed in a block 70 suspended from top portion 42 of suspension bracket 40. It should be noted that Fig. 5 omits top portion 42 of suspension bracket 40 in order to show a tip stop mechanism 170 explained in greater detail below.

[0021] In addition to dove-tail shaped projections 72 and grooves 74, the present invention contemplates the use of different shapes for projections 72 and grooves 74. Moreover, instead of projections 72 and grooves 74 provided, respectively, on cutting heads 28 and blocks 70, the present invention contemplates the use of grooves 74 on cutting heads 28 and corresponding projections 72 on blocks 70.

[0022] To move each cutting head 28 in lateral directions designated by reference number 66 (Fig. 1), a stepper motor 60 is mounted to each side portion 44 of suspension bracket 40. Accordingly, stepper motor 60 moves longitudinally with suspension bracket 40 as indicated by arrow 46 in Fig. 2. A threaded shaft 62 (Figs. 7 and 14) extends through side portion 44 of suspension bracket 40 and is connected to a block 64 protruding from the front surface of each cutting head 28. Similar to threaded shaft 50, threaded shaft 62 moves linearly as the motor armature (not shown) of stepper motor 60 rotates in first and second directions. Through the connection to block 64, threaded shaft 62 pushes or pulls cutting head 28. In response to the linear movements of threaded shaft 62, cutting head 28 moves by sliding dove-tail shaped projection 72 in groove 74.

[0023] While cutting heads 28 move longitudinally and laterally to cut a desired notch pattern on key blank 33, a clamp mechanism 80 holds key blank 33 stationary. Clamp mechanism 80 includes a lower clamp or anvil 82 and an upper clamp 84. Anvil 82 is supported on base 23 and includes a recess 86 formed on the top portion thereof as shown in Fig. 6. Anvil 82 further includes a pair of pins 88 protruding from the top portion thereof toward top frame 24. Upper clamp 84 includes a pair of holes (not shown) that correspond to and engage pins 88. Upper clamp 84 is movable vertically to and away from anvil 82. Pins 88 ensure upper clamp 84 moves in a substantially vertical direction.

[0024] As illustrated in Figs. 4, 11 and 18, anvil 82 includes an elongated portion 83 configured to support a blade portion 35 of key blank 33. Elongated portion 83 is positioned at the lateral center of anvil 82 behind recess 86. Elongated portion 83 has a length to support the entire length of blade portion 35 of key blank 33 and a width narrower than the width of blade portion 35 of key blank 33. The top of elongated portion 83 is coplanar with the bottom of recess 86. Similarly, upper clamp 84 includes a corresponding elongated portion 85 (Figs. 4 and 18)

configured to press down on blade portion 35 of key blank 33. Elongated portions 83 and 85 provide support for the cutting area of blade portion 35 to minimize the rotational leverage (i.e., torque) applied to key blank 33 by cutting wheels 29. Preferably, clamping mechanism 80 does not engage the handle of key blank 33. Thus, the clamp does not have to account for keys where the handle thickness is different than the blade thickness, as is the case with plastic handled key.

[0025] As illustrated in Fig. 11, the bottom of recess 86 and elongated portion 83 combine to form a T-shaped supporting surface. Similarly, the bottom surface 95 (Figs. 1 and 18) of upper clamp 84 and elongated portion 85 combine to form a T-shaped pressing surface. The bottom of recess 86 and the bottom surface 95 of upper clamp 84 are configured to support and press the entire width of a shank portion 39 of key blank 33. By supporting and pressing the entire width of shank portion 39, the bottom of recess 86 and the bottom surface 95 prevent key blank 33 from tilting even if elongated portions 83 and 85 engage a groove 41 or other uneven milling surfaces.

[0026] To move upper clamp 84 to and away from anvil 82, clamp mechanism 80 utilizes a cam 90 and springs 92. As illustrated in Figs. 3, 4, 15, 19, and 20, clamp mechanism 80 include a support block 100 secured to top frame 24 and a block 98 shaped like an upside down T. Support block 100 includes an elongated recess 102 and a hole 104 extending through the top portion 101 thereof. Block 98 includes a pin 99 protruding from the top portion 97 thereof. Block 98 is positioned in elongated recess 102 of support block 100. The height of elongated recess 102 is greater than that of block 98 so that block 98 is movable vertically between top portion 101 of support block 100 and the upper surface of top frame 24. Pin 99 received through hole 104 of support block 100 guides the vertical movements of block 98.

[0027] As illustrated in Fig. 3, block 98 includes holes 106 configured to receive bolts 91 therethrough. Bolts 91 received through holes 106 of block 98 are secured to top frame 24. Block 98 is connected to a clamp screw 94 extending through a hole 96 in top frame 24. Clamp screw 94 in turn is connected to upper clamp 84. Springs 92, secured by bolts 91, exert a downward biasing force on block 98, and therefore, on clamp screw 94. The spring force acts on block 98 to move upper clamp 84 toward anvil 82 and into a closed position. Depending on the thickness of key blank 33, the precise position of the bottom surface 95 of upper clamp 84 in a closed position varies. Springs 92, however, provides a sufficient downward force to hold key blank 33 stationary regardless of the precise position of the bottom surface 95 of upper clamp 84.

[0028] To overcome the downward biasing force of springs 92 and to move upper clamp 84 upward to an open position, clamp mechanism 80 utilizes cam 90 and a shaft 110. Shaft 110 extends parallel to top frame 24 through holes 112 of support block 100 and a hole 114

of block 98. Bearings 116 mounted in holes 112 of support block 100 rotatably support shaft 110. Shaft 110 is rotated by a DC motor 118. DC motor 118 is mounted to a block 120 extending from top frame 24 and includes a rotary shaft 122 coupled to shaft 110 by a coupler 124.

[0029] As illustrated in Figs. 9 and 10, cam 90 is mounted on an eccentric portion 111 of shaft 110 and rotates with shaft 110. Cam 90 is press-fitted onto eccentric portion 111 of shaft 110 so that the center 125 of cam 90 is offset from the rotational axis 123 of shaft 110. Accordingly, a portion 132 is positioned farther away from rotational axis 123 of shaft 110 than a portion 130. Preferably, cam 90 includes a roller bearing 127 so that it rolls rather than slides against the inner surface of opening 114. When DC motor 118 rotates shaft 110 one hundred eighty degrees from an open position, as illustrated in Fig. 10, to a closed position, as illustrated in Fig. 9, portion 132 of cam 90 disengages the upper surface of opening 114. The downward biasing force exerted by springs 92 then moves block 98 toward the upper surface of top frame 24. As block 98 moves toward the upper surface of top frame 24, it transmits the downward biasing force exerted by springs 92 to clamp screw 94. Clamp screw 94 in turn transmits the downward force exerted by springs 92 to upper clamp 84 to hold key blank 33. As mentioned above, however, the precise position of the bottom surface 95 of upper clamp 84 in a closed position depends on the thickness of key blank 33. Accordingly, when upper clamp 84 is in a closed position, the size of a gap 133 developed between the bottom surface of block 98 and the upper surface of top frame 24 varies depending on the thickness of key blank 33.

[0030] On the other hand, when DC motor 118 rotates shaft 110 one hundred eighty degrees from a closed position, as illustrated in Fig. 9, to the open position, as illustrated in Fig. 10, portion 132 of cam 90 contacts the upper surface of opening 114. The contact between portion 132 of cam 90 and the upper surface of opening 114 then overcomes the downward biasing force exerted by springs 92 and moves block 98 toward upper portion 101 of supporting block 100. As block 98 moves toward upper portion 101 of support block 100, it pulls clamp screw 94 upward. Clamp screw 94 in turn pulls upper clamp 84 upward to release key blank 33.

[0031] Preferably, cam 90 is circular in shape. However, cam 90 may assume many different shapes, including but not limited to ellipses, ovals, or any combination thereof, as long as the distances between rotational axis 123 of shaft 110 and portions 130 and 132 are different. Alternatively, instead of eccentric portion 111, shaft 110 itself may provide the necessary eccentricity. For example, the portion of shaft 110 positioned within opening 114 of block 98 may be offset from the rotational axis 123 of shaft 110 to provide the necessary eccentricity.

[0032] Two position switches 140 and 142 (Figs. 3 and 15) mounted on the front surface of support block 100 control the rotation of DC motor 118 between the open and closed positions. When shaft 110 reaches the closed

position, it activates position switch 140 and stops DC motor 118. Similarly, when shaft 110 reaches the open position, it activates position switch 142 and stops DC motor 118. Two position switches 140 and 142 are spaced apart from each other by one hundred eighty degrees. To selectively activate one of two switches 140 and 142, the front end of shaft 110 protruding from the front surface of support block 100 includes a projection 144. Projection 144 will rotate into engagement with one of the two position switches after each 180° of rotation.

[0033] As described above, no component of clamp mechanism 80, except for anvil 82, is supported on base 23. The present invention, however, contemplates an alternative clamp mechanism where every component, including an anvil, is suspended from the top of key cutting machine 20. In one embodiment of an alternative clamp mechanism, an anvil having a hollow portion may be suspended from top frame 24 and upper clamp 84 described above may move vertically within the hollow portion. By suspending the anvil, key cutting machine 20 eliminates the need for base 23 and further reduces the possibility of cutting chips pilling up and contaminating components of key cutting machine 20.

[0034] To laterally position key blank 33, each key cutting head 28 includes an alignment structure or a paddle 150, as illustrated in Figs. 5, 6, and 17. Each paddle 150 includes a key guide portion 151 having a groove 152. Groove 152 of key guide portion 151 is positioned to be coplanar with the bottom of recess 86 so that it engages a side edge of blade portion 35 of key blank 33. Furthermore, a pin 154 pivotally mounts paddle 150 to cutting head 28 and a spring 156 biases key guide portion 151 of paddle 150 away from the rotational axis of cutting wheel 29. An adjustment screw 157 mounted on each cutting head 28 adjusts the precise lateral position of key guide portion 151 with respect to the rotational axis of cutting wheel 29.

[0035] Accordingly, when stepper motors 60 laterally position cutting heads 28 so that the distance between key guide portions 151 of paddles 150 correspond to the width of blade portion 35 of key blank 33, grooves 152 engage and guide key blank 33 into a correct lateral position. Furthermore, springs 126 automatically correct any deviation from the correct lateral position. It should be noted that Figs. 5 and 6 show groove 152 of key guide portion 151 above key blank 33 solely for the purpose of illustrating the construction details of paddle 150. As described above, groove 152 is coplanar with the bottom of recess 86 unlike its position shown in Figs. 5 and 6.

[0036] Key guide portion 151 of paddle 150 is positioned slightly behind cutting wheel 29 and is aligned tangent to the edge of cutting wheel 29 by adjustment screw 157. From this aligned position, key guide portion 151 is movable to the rotational axis of cutting wheel 29 behind cutting wheel 29 and never contacts the edge of cutting wheel 29. As cutting head 28 moves laterally toward key blank 33, key guide portion 151 collapses back toward the rotational axis of cutting wheel 29 and overcomes the

biasing force exerted by spring 156. Therefore, key guide portion 151 of paddle 150 pivots around pin 154 and moves toward the rotational axis of cutting wheel 29 behind the edge of cutting wheel 29. As a result, the edge of cutting wheel 29 contacts and cuts a side edge of blade portion 35 slightly in front of key guide portion 151 of paddle 150. On the other hand, as cutting head 28 moves laterally away from key blank 33, key guide portion 151 pivots away from the rotational axis of cutting wheel 29 toward key blank 33 by the biasing force exerted by spring 156 until it again is aligned tangent to the edge of cutting wheel 29.

[0037] To longitudinally position key blank 33, key cutting machine 20 includes a tip stop mechanism 170 illustrated in Figs. 7, 8, and 12. Tip stop mechanism 170 includes a stepper motor 172 mounted to a block 174 supported on base 23. Instead of being supported on base 23, block 174 may be suspended from top frame 24. A threaded lead screw 176 extends through stepper motor 172 and one end thereof is connected to a tip stop 178. The front end of tip stop 178 is configured to engage the tip 37 (Fig. 11) of key blank 33. Similar to stepper motors 48 and 60 explained above, stepper motor 172 rotates threaded lead screw 176 in first and second directions to move tip stop 178 longitudinally to and away from key blank 33.

[0038] Tip stop mechanism 170 further includes a microswitch 180 to indicate the longitudinal home position of tip stop 178. When the back end of threaded lead screw 176 activates microswitch 180, key cutting machine 20 senses that tip stop 178 is in its longitudinal home position. From this longitudinal home position, key cutting machine 20 determines the amount of longitudinal movement that tip stop 178 needs to travel to position key blank 33 in a correct longitudinal position. Of course, the amount of longitudinal movement from the longitudinal home position varies depending on the type of key blank 33.

[0039] Similarly, stepper motors 60 have microswitches (not shown) to indicate the lateral home positions of cutting heads 28. From these lateral home positions, key cutting machine 20 determines the amount of lateral movement that each cutting head 28 needs to travel to position each paddle 150 in a correct lateral position and to cut blade portion 35 of key blank 33 to a desired depth. In addition, stepper motor 48 has a microswitch (not shown) to indicate the longitudinal home position of cutting heads 28. From this longitudinal home position, key cutting machine 20 determines the amount of longitudinal movement that cutting heads 28 need to travel in unison to cut a desired notch pattern on blade portion 35 of key blank 33.

[0040] Key cutting machine 20 eliminates the need for the cassette described in U.S. Patent No. 5,676,504. The cassette described in '504 patent provides both positional alignment function and clamping function. As described above, however, paddles 150 and tip stop mechanism 170 perform the positional alignment function of the cas-

sette described in '504 patent. Furthermore, clamping mechanism 80 performs the clamping function of the cassette described in '504 patent. Therefore, key cutting machine 20 does not require the cassette described in '504 patent.

[0041] Key cutting machine 20 cuts a desired notch pattern on key blank 33 without reference to or tracing of the notch pattern on a master key. Rather, key cutting machine 20 cuts key blank 33 based on a series of code numbers defining a desired notch pattern. In addition to a series of code numbers defining a desired notch pattern, key cutting machine 20 needs relevant data (e.g., shape, length and width of blade, etc.) corresponding to the correct key blank. An operator may manually determine the correct key blank and input into a control panel (not shown) a unique identifier corresponding to the correct key blank. Based on the unique identifier entered, key cutting machine 20 retrieves the relevant information from its memory or database (not shown). As alternative to an operator manually determining the correct key blank, key cutting machine 20 may be connected to a key identification system, such as the system described in copending U.S. Application No. 09/625,274 to identify the correct key blank. The entire disclosure of copending U.S. Application No. 09/625,274 is hereby incorporated by reference. It is contemplated that other types of key identification systems may also be used.

[0042] The operation of the aforementioned key cutting machine 20 will now be described with reference to the attached drawings.

[0043] When an operator turns on or resets key cutting machine 20, cutting heads 28 move to their lateral and longitudinal home positions and tip stop 178 moves to its longitudinal home position. The type of key blank 33 and its unique identifier are then determined either by the operator manually or by a key identification system, such as the system described in copending U.S. Application No. 09/625,274. Based on the unique identifier entered, key cutting machine 20 retrieves the relevant information (e.g., shape, length and width of blade, etc.) about key blank 33 from its memory or database. In addition to the unique identifier, a series of code numbers defining a desired notch pattern is entered into key cutting machine 20. Alternatively, a notch pattern corresponding exactly to a captured image of a master key may be entered instead of a code defining the notch pattern. At this point, stepper motors 60 move cutting heads 28 laterally so that key guide portions 151 of paddles 150 are spaced apart by the distance corresponding to the width of key blank 33. Cutting heads 28 and tip stop 178, however, stay in their respective longitudinal home positions.

[0044] Depending on the type of key blank 33 to be cut, the lateral positions of cutting heads 28 may be symmetric or offset with respect to a vertical center line 190 (Fig. 3) of key cutting machine 20. For example, the lateral positions of cutting heads 28 are symmetric with respect to vertical center line 190 if a double-sided key blank is to be cut. On the other hand, the lateral positions

of cutting heads 28 are offset to the left or to the right of vertical center line 190 if a single-sided key blank is to be cut. For a single-sided key blank, key cutting machine 20 may be programmed to use either one of two cutting heads 28 and the operator needs to insert the single-sided key blank in a correct orientation. For example, if key cutting machine 20 is programmed to use cutting head 28 to the right of vertical center line 190, the operator needs to insert a single-sided key blank so that the side edge to be cut faces to the right. To assist the operator in correctly inserting single-sided key blanks, key cutting machine 20 may provide instructions in its control panel or have an instruction label affixed on the front panel thereof.

[0045] After moving cutting heads 28 to their correct lateral positions, key cutting machine 20 asks the operator to insert key blank 33. When the operator inserts key blank 33 through the opening formed between recess 86 of anvil 82 and the bottom surface 95 of upper clamp 84, grooves 152 of key guide portions 151 of paddles 150 engage the side edges of key blank 33 and guide key blank 33 into its correct lateral position. Springs 156 biasing key guide portions 151 of paddles 150 away from the rotational axes of cutting wheels 29 automatically correct any lateral misalignment introduced by the operator.

[0046] The operator stops inserting key blank 33 when tip 37 of key blank 33 engages the front end of tip stop 178 or when the shoulder of key blank 33 engages the front surface of anvil 82. Then, the operator initiates the key cutting process. At this point, key blank 33 is in its correct lateral position but may not be in its correct longitudinal position because tip stop 178 is in its longitudinal home position. However, when the operator initiates the key cutting process, stepper motor 172 moves tip stop 178 to its correct longitudinal position. During its course of movement, tip stop 178 engages tip 37 of key blank 33 and moves key blank 33 to its correct longitudinal position.

[0047] After stepper motor 172 moves tip stop 178 to its correct longitudinal position, DC motor 118 rotates shaft 110 from its open position to its closed position, thereby allowing springs 92 to push upper clamp 84 down toward key blank 33. Under the biasing force of springs 92 and its own weight, upper clamp 84 presses down on key blank 33, which is supported on its opposite side by the bottom of recess 86 and elongated portion 83 of anvil 82. Preferably, tip stop 178 has a thickness less than any key blank to be cut by key cutting machine 20. Therefore, elongated portions 83 and 85 do not clamp tip stop 178, even when clamping the thinnest of key blanks.

[0048] After clamping key blank 33 in its correct lateral and longitudinal positions, key cutting machine 20 moves cutting heads 28 laterally and longitudinally to cut a desired notch pattern on key blank 33 according to the series of code numbers entered. Alternatively, cutting heads 28 may cut a notch pattern based on a captured image, such as a digital or video image, of a master key. In other words, cutting heads 28 may cut either a notch

pattern corresponding exactly to a captured image of a master key or a notch pattern corresponding to an original specification based on the captured image of the master key. In the latter case, key cutting machine 20 reproduces a notch pattern corresponding to the original specification even if the master key has worn down because of repeated use. While cutting heads 28 move longitudinally in unison, they move independently in lateral directions. Accordingly, during the key cutting process, key cutting machine 20 moves both cutting heads 28 laterally if a double-sided key blank 33 is inserted. However, key cutting machine 20 moves only one of cutting heads 28 laterally during the key cutting process if a single-sided key blank 33 is inserted. The other cutting head 28, on the other hand, does not move laterally during the key cutting process. Before the key cutting process starts, key cutting machine 20 retracts the other cutting head 28 away from a single-sided key blank 33.

[0049] After cutting a desired notch pattern, DC motor 118 rotates shaft 110 from its closed position to its open position. Then, upper clamp 84 moves up to allow the operator to remove key blank 33 from key cutting machine 20.

[0050] In accordance with the present invention, another embodiment of a key cutting machine is illustrated in Figs. 21-23. Clamp mechanism 80 and a housing including top frame 24 and side frame 26 are omitted in Fig. 21 for the purpose of illustrating inner details. Likewise, a portion of clamp mechanism 80, side frame 26, and top portion 42 of suspension bracket 40 are omitted in Figs. 22 and 23. Many aspects of a key cutting machine 220 are identical to key cutting machine 20 illustrated in Figs. 1-20. However, unlike each cutting head 28 in key cutting machine 20, which moves laterally in a linear motion, each cutting head 28 in key cutting machine 220 moves laterally in a pivoting motion as designated by reference number 222.

[0051] To move each cutting head 28 laterally in a pivoting motion, each cutting head 20 includes a pivot arm 224. Each pivot arm 224 includes a pivot rod 226, which is pivotally mounted to a corresponding pair of bearing supports 238 through sleeve bearings 228. Each pivot arm 224 extends through top portion 42 of suspension bracket 40 through an opening 230. Each pivot arm 224 is, therefore, pivotally suspended from a corresponding pair of bearing supports 238. Bearing supports 238 are connected to top portion 42 of suspension bracket 40. Accordingly, both cutting heads 28, including their corresponding pivot arms 224, move longitudinally in unison as suspension bracket 40 moves along guide rails 34 in longitudinal directions. Each cutting head 28, however, independently moves laterally in a pivoting motion.

[0052] Each cutting head 28 includes a push block 232 and a push rod 234. Each push rod 234 is connected to threaded shaft 62 of stepper motor 60 and abuts a corresponding push block 232 extending from a bottom portion of a corresponding cutting head 28. Each cutting head 28 further includes a spring 236 (Fig. 23) connected

at one end to a corresponding push block 232 and at the other end to a corresponding side portion 44 of suspension bracket 40. Each spring 236 biases a corresponding push block 232 toward a corresponding side portion 44 of suspension bracket 40. In other words, each spring 236 biases a corresponding push block 232 away from clamp mechanism 80. As each push rod 234 moves linearly away from clamp mechanism 80, a corresponding spring 236 pulls a corresponding push block 232 and pivots a corresponding pivot arm 224 away from clamp mechanism 80. Accordingly, each cutting head 28 pivots away from clamp mechanism 80 as push rod 234 moves linearly away from clamp mechanism 80. Conversely, as each push rod 234 moves linearly toward clamp mechanism 80, a corresponding push rod 234 pushes a corresponding push block 232 and overcomes the biasing force exerted by a corresponding spring 236. Accordingly, a corresponding pivot arm 224 and a corresponding cutting head 28 pivot toward clamp mechanism 80 as each push rod 234 moves linearly toward clamp mechanism 80.

[0053] This arrangement of push rod 234 abutting against push block 232 biased by spring 236 provides an "anti-backlash" feature. Because push rod 234 pushes against the biasing force exerted by spring 236 on push block 232, push rod 234 and push block 232 are always in contact with each other. In other words, even if the contact surfaces between push rod 234 and push block 232 are worn down or dented, no gap develops between push rod 234 and push block 232. Accordingly, the lateral movements of cutting heads 28 are unaffected by worn-down or dented contact surfaces. Without this "anti-backlash" feature, a gap developed between worn-down or dented engaging surfaces may compromise accurate lateral movements of cutting heads 28.

[0054] Aside from the lateral movement of each cutting head 28 in a pivoting motion, the operation of key cutting machine 220 is identical to that of key cutting machine 20.

[0055] It will be apparent to those skilled in the art that various modifications and variations can be made in the device of the present invention without departing from the scope of the invention. Other embodiments of the invention will be apparent to those skilled in the art from consideration of the specification and practice of the invention disclosed herein.

Claims

1. A key cutting machine (20, 220) for cutting a notch pattern on a key blank (33), comprising a housing and a pair of cutting heads (28), each cutting head including a cutting wheel (29) rotatable to cut the notch pattern on the key blank, wherein the housing includes a top frame (24); the cutting heads are positioned within the housing; and the key cutting machine further comprises:

a clamp mechanism (80) positioned between the cutting heads, the clamp mechanism including a lower clamp (82) and an upper clamp (84), the upper clamp being suspended from the top frame of the housing and vertically movable relative to the lower clamp to hold the key blank stationary, and the lower clamp having a front surface for engaging a shoulder of the key blank when the key blank is inserted; **characterized in that:**

each cutting head further includes a paddle (150) mounted behind a corresponding cutting wheel and the paddle has a key guide portion (151) aligned tangent to an edge of the corresponding cutting wheel to engage and position the key blank laterally.

2. The key cutting machine of claim 1, wherein the housing further includes a base (23) and the lower clamp is supported on the base of the housing.
3. The key cutting machine of claim 2, wherein the lower clamp includes a recess (86) configured to support a shank portion (39) of the key blank and an elongated portion (83) configured to support a blade portion (35) of the key blank.
4. The key cutting machine of claim 3, wherein the upper clamp includes a bottom surface (95) configured to press the shank portion of the key blank and an elongated portion (85) configured to press the blade portion of the key blank.
5. The key cutting machine of claim 1, wherein the clamp mechanism further includes a block (100) positioned above the top frame of the housing and a clamp screw (94) extending through the top frame of the housing and connected to the block and the upper clamp.
6. The key cutting machine of claim 5, wherein the block is vertically movable between an open position away from the top frame of the housing and a closed position toward the top frame of the housing.
7. The key cutting machine of claim 6, wherein the clamp mechanism further includes a pair of springs (92) biasing the block toward the closed position.
8. The key cutting machine of claim 7, wherein the block includes a hole (112) extending therethrough and the clamp mechanism further includes a shaft (110) and a cam (90) eccentrically mounted on the shaft in the hole of the block to move the block between the open and closed positions.
9. The key cutting machine of claim 8, wherein the block

is movable between the open and closed positions for every 180° rotation of the shaft.

10. The key cutting machine of claim 9, wherein the clamp mechanism further includes a pair of position switches (140, 142) spaced apart from each other by 180° and the shaft includes a projection (144) engageable with the position switches for every 180° rotation of the shaft. 5
11. The key cutting machine of claim 1, wherein each cutting head further includes a spring (156) biasing the key guide portion of the paddle away from a rotational axis of a corresponding cutting wheel. 10
12. The key cutting machine of claim 11, wherein the key guide portion of each paddle is pivotable toward the rotational axis of a corresponding cutting wheel. 15
13. The key cutting machine of claim 1, wherein the clamp mechanism is stationary and the cutting heads are movable relative to the clamp mechanism. 20
14. The key cutting machine of claim 1, further comprising a tip stop mechanism (170) positioned behind the clamp mechanism and configured to engage and position the key blank longitudinally. 25
15. The key cutting machine of claim 1 wherein: 30
 - the housing includes a base (23);
 - the cutting heads (28) are suspended below the top frame (24) of the housing, and
 - the cutting heads (28) are moveable longitudinally in unison, each cutting head (28) being connected to a stepper motor (60) for actively moving the cutting head in lateral directions, perpendicular to the longitudinal direction of key blank insertion. 35
16. The key cutting machine of claim 15, further comprising a pair of guide rails (34) suspended from the top frame of the housing, a pair of bearing supports (38, 238) slidably mounted to each guide rail, and a carriage (40) connected to the bearing supports. 40
17. The key cutting machine of claim 15, wherein the cutting heads (28) are connected to a suspension bracket (40) suspended below the top frame (24) of the housing. 45

Patentansprüche

1. Schlüsselfräsmaschine, (20, 220) zum Fräsen eines Kerbmusters auf einen Schlüsselrohling (33), umfassend ein Gehäuse und ein Paar von Fräsköpfen (28), wobei jeder Fräskopf ein Fräsrads (29) ein- 55

schließt, das drehbar ist, um das Kerbmuster auf den Schlüsselrohling zu fräsen, wobei:

das Gehäuse einen oberen Rahmen (24) einschließt;
die Fräsköpfe innerhalb des Gehäuses positioniert sind; und
die Schlüsselfräsmaschine ferner umfasst:

einen Spannmechanismus (80), der zwischen den Fräsköpfen positioniert ist, wobei der Spannmechanismus eine untere Spannklemme (82) und eine obere Spannklemme (84) einschließt, wobei die obere Spannklemme am oberen Rahmen des Gehäuses angehängt ist und vertikal relativ zu der unteren Spannklemme beweglich ist, um den Schlüsselrohling stationär zu halten, und wobei die untere Spannklemme eine Frontfläche zum Erfassen einer Schulter des Schlüsselrohlings aufweist, wenn der Schlüsselrohling eingesetzt wird;

dadurch gekennzeichnet, dass jeder Fräskopf ferner einen Flügel (150) einschließt, der hinter einem korrespondierenden Fräsrads montiert ist, und wobei der Flügel einen Schlüsselführungsbereich (151) aufweist, der tangential zu einem Rand des korrespondierenden Fräsrads ausgerichtet ist, um den Schlüsselrohling lateral zu erfassen und zu positionieren.

2. Schlüsselfräsmaschine nach Anspruch 1, wobei das Gehäuse ferner eine Basis (23) einschließt und die untere Spannklemme an der Basis des Gehäuses abgestützt ist. 35
3. Schlüsselfräsmaschine nach Anspruch 2, wobei die untere Spannklemme eine Ausnehmung (86), die konfiguriert ist, um einen Schaftbereich (39) des Schlüsselrohlings zu stützen, und einen länglichen Bereich (83), der konfiguriert ist, um einen Klingensbereich (35) des Schlüsselrohlings zu stützen einschließt. 40
4. Schlüsselfräsmaschine nach Anspruch 3, wobei das obere Spannelement eine Bodenfläche (95), die konfiguriert ist, um den Schaftbereich des Schlüsselrohlings zu pressen, und einen länglichen Bereich (85), der konfiguriert ist, um den Klingensbereich des Schlüsselrohlings zu pressen, einschließt. 50
5. Schlüsselfräsmaschine nach Anspruch 1, wobei der Spannmechanismus ferner einen Block (100), der oberhalb des oberen Rahmen des Gehäuses positioniert ist, und eine Spannschraube (94), die sich durch den oberen Rahmen des Gehäuses erstreckt

und mit dem Block und dem oberen Spannelement verbunden ist, einschließt.

6. Schlüsselfräsmaschine nach Anspruch 5, wobei der Block vertikal zwischen einer offenen Position weg von dem oberen Rahmen des Gehäuses und einer geschlossenen Position in Richtung auf den oberen Rahmen des Gehäuses bewegbar ist. 5
7. Schlüsselfräsmaschine nach Anspruch 6, wobei der Klemmmechanismus ferner ein Paar von Federn (92) einschließt, die den Block in Richtung auf die geschlossene Position spannen. 10
8. Schlüsselfräsmaschine nach Anspruch 7, wobei der Block eine Öffnung (112) einschließt, die sich hindurch erstreckt, und wobei der Spannmechanismus ferner einen Schaft (110) und eine Nocke (90), die exzentrisch auf dem Schaft in der Öffnung des Blocks montiert ist, einschließt, um den Block zwischen den offenen und geschlossenen Positionen zu bewegen. 15 20
9. Schlüsselfräsmaschine nach Anspruch 8, wobei der Block zwischen den offenen und geschlossenen Positionen für jede 180°-Drehung des Schafts bewegbar ist. 25
10. Schlüsselfräsmaschine nach Anspruch 9, wobei der Spannmechanismus ferner ein Paar von Positionsschaltern (140, 142) einschließt, die voneinander um 180° beabstandet sind, und wobei der Schaft einen Vorsprung (144) einschließt, der mit den Positionsschaltern für jede 180°-Drehung des Schafts in Eingriff bringbar ist. 30 35
11. Schlüsselfräsmaschine nach Anspruch 1, wobei jeder Fräskopf ferner eine Feder (156) einschließt, die den Schlüsselführungsbereich des Flügels von einer Drehachse eines korrespondierenden Fräsrads weg vorspannt. 40
12. Schlüsselfräsmaschine nach Anspruch 11, wobei der Schlüsselführungsbereich jedes Flügels in Richtung auf die Drehachse eines korrespondierenden Fräsrads drehbar ist. 45
13. Schlüsselfräsmaschine nach Anspruch 1, wobei der Spannmechanismus stationär ist und die Fräsköpfe relativ zu dem Spannmechanismus bewegbar sind. 50
14. Schlüsselfräsmaschine nach Anspruch 1, weiter umfassend einen Spitzenstopmechanismus (170), der hinter dem Spannmechanismus positioniert ist und konfiguriert ist, um den Schlüsselrohling longitudinal in Eingriff zu bringen und zu positionieren. 55
15. Schlüsselfräsmaschine nach Anspruch 1, wobei:

das Gehäuse eine Basis (23) einschließt; die Fräsköpfe (28) unter dem oberen Rahmen (24) des Gehäuses aufgehängt sind; und die Fräsköpfe (28) longitudinal übereinstimmend bewegbar sind, wobei jeder Fräskopf (28) mit einem Schrittmotor (60) zum aktiven Bewegen des Fräskopf in lateralen Richtungen verbunden ist, senkrecht zu der longitudinalen Richtung des Schlüsselrohlingeinsatzes.

16. Schlüsselfräsmaschine nach Anspruch 15, weiter umfassend ein Paar von Führungsschienen (34), die an dem oberen Rahmen des Gehäuses aufgehängt sind, ein Paar von Lagerträgern (38, 238), die leitfähig auf jeder Führungsschiene montiert sind, und ein Fahrgestell (40), das mit dem Lagerträgern verbunden ist.
17. Schlüsselfräsmaschine nach Anspruch 15, wobei die Fräsköpfe (28) mit einer Aufhängeklammer (40) verbunden sind, die unter dem oberen Rahmen (24) des Gehäuses aufgehängt ist.

Revendications

1. Découpeuse de clés (20, 220) servant à découper une série d'encoches dans une pièce brute de clé (33), comportant un carter et deux têtes de découpage (28), chacune de ces têtes de découpage comprenant une roue de découpage (29) qui peut être mise en rotation pour découper une série d'encoches dans la pièce brute de clé, **caractérisée en ce que:**

le carter comprend un bâti supérieur (24);
les têtes de découpage se trouvent positionnées à l'intérieur du carter; et
la découpeuse de clés comporte par ailleurs:

un mécanisme de serrage (80) positionné entre les têtes de découpage, le mécanisme de serrage comprenant un serrage inférieur (82) et un serrage supérieur (84), le serrage supérieur étant suspendu à partir du bâti supérieur du carter et pouvant être déplacé verticalement par rapport au serrage inférieur de manière à tenir la clé en position fixe, et le serrage inférieur ayant une surface avant qui sert à entrer en contact avec un épaulement de la pièce brute de clé lorsque celle-ci est introduite; **caractérisée en ce que**

chaque tête de découpage comporte par ailleurs une palette (150) montée derrière une roue de découpage correspondante, et la palette comporte

une partie servant de guide-clé (151) alignée tangentiellement par rapport à un bord de la roue de découpage correspondante afin d'entrer en contact avec la pièce brute de clé et la positionner latéralement.

2. Découpeuse de clés selon la revendication 1, **caractérisée en ce que** le carter comporte par ailleurs une base (23) et **en ce que** le serrage inférieur est soutenu sur la base du carter. 5
3. Découpeuse de clés selon la revendication 2, **caractérisée en ce que** le serrage inférieur comporte un évidement (86) qui est configuré de manière à soutenir le corps (39) de la pièce brute de clé, et aussi une partie allongée (83) qui est configurée de manière à soutenir la tige (35) de la pièce brute de clé. 10
4. Découpeuse de clés selon la revendication 3, **caractérisée en ce que** le serrage supérieur comprend une surface inférieure (95) qui est configurée de manière à serrer le corps de la pièce brute de clé, et aussi une partie allongée (85) qui est configurée de manière à serrer la tige de la pièce brute de clé. 15
5. Découpeuse de clés selon la revendication 1, **caractérisée en ce que** le mécanisme de serrage comprend par ailleurs un bloc (100) qui est positionné au-dessus du bâti supérieur du carter et aussi une vis de serrage (94) s'étendant à travers le bâti supérieur du carter et raccordée au bloc et au serrage supérieur. 20
6. Découpeuse de clés selon la revendication 5, **caractérisée en ce que** le bloc peut être déplacé verticalement entre une position d'ouverture éloignée du bâti supérieur du carter et une position de fermeture vers le bâti supérieur du carter. 25
7. Découpeuse de clés selon la revendication 6, **caractérisée en ce que** le mécanisme de serrage comprend par ailleurs deux ressorts (92) servant à attirer le bloc vers la position de fermeture. 30
8. Découpeuse de clés selon la revendication 7, **caractérisée en ce que** le bloc comprend un trou (112) qui le traverse de part en part, et **en ce que** le mécanisme de serrage comporte par ailleurs un arbre (110) et une came (90) montée de manière excentrique sur l'arbre dans le trou du bloc afin de déplacer le bloc entre les positions d'ouverture et de fermeture. 35
9. Découpeuse de clés selon la revendication 8, **caractérisée en ce que** le bloc peut être déplacé entre les positions d'ouverture et de fermeture chaque fois 40

que l'arbre tourne d'un angle de 180°.

10. Découpeuse de clés selon la revendication 9, **caractérisée en ce que** le mécanisme de serrage comporte par ailleurs deux contacteurs de position (140, 142) espacés de 180° l'un par rapport à l'autre, et **en ce que** l'arbre comprend un élément en saillie (144) qui entre en contact avec les contacteurs de position chaque fois que l'arbre tourne d'un angle de 180°. 45
11. Découpeuse de clés selon la revendication 1, **caractérisée en ce que** chaque tête de découpage comporte par ailleurs un ressort (156) qui a pour effet d'éloigner, d'un axe de rotation d'une roue de découpage correspondante, la partie servant de guide-clé de la palette. 50
12. Découpeuse de clés selon la revendication 11, **caractérisée en ce que** la partie servant de guide-clé de chaque palette peut pivoter vers l'axe de rotation d'une roue de découpage correspondante. 55
13. Découpeuse de clés selon la revendication 1, **caractérisée en ce que** le mécanisme de serrage est fixe tandis que les têtes de découpage peuvent se déplacer relativement au mécanisme de serrage.
14. Découpeuse de clés selon la revendication 1, comportant par ailleurs un mécanisme de butée de l'extrémité (170) placé derrière le mécanisme de serrage et configuré de manière à entrer en contact avec la pièce brute de clé et la positionner longitudinalement.
15. Découpeuse de clés selon la revendication 1, **caractérisée en ce que**:
 le carter comprend une base (23);
 les têtes de découpage (28) sont suspendues au-dessous du bâti supérieur (24) du carter; et les têtes de découpage (28) peuvent être déplacées ensemble longitudinalement, chaque tête de découpage (28) étant raccordée à un moteur pas à pas (60) qui sert à déplacer activement la tête de découpage latéralement, et à angle droit par rapport au sens longitudinal d'insertion de la pièce brute de clé.
16. Découpeuse de clés selon la revendication 15, comportant par ailleurs deux rails de guidage (34) suspendus à partir du bâti supérieur du carter, deux supports de palier (38, 238) montés sur chaque rail de guidage et pouvant coulisser sur celui-ci, et un chariot (40) raccordé aux supports de palier.
17. Découpeuse de clés selon la revendication 15, **caractérisée en ce que** les têtes de découpage (28)

sont raccordées à un support de suspension (40) suspendu au-dessous du bâti supérieur (24) du carter.

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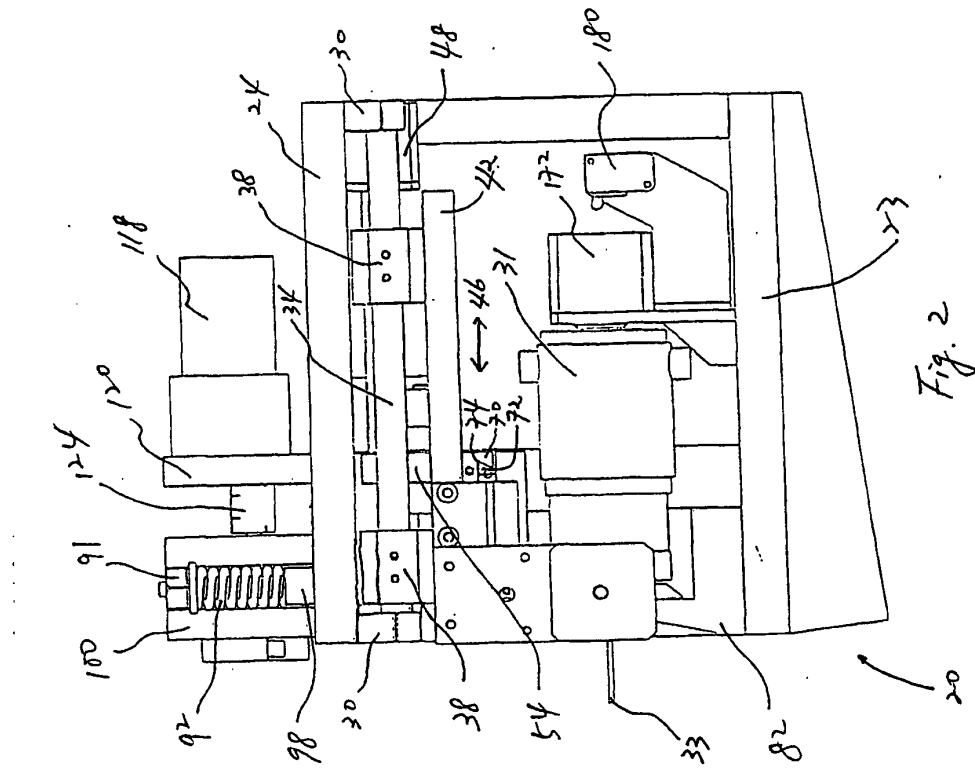


Fig. 1

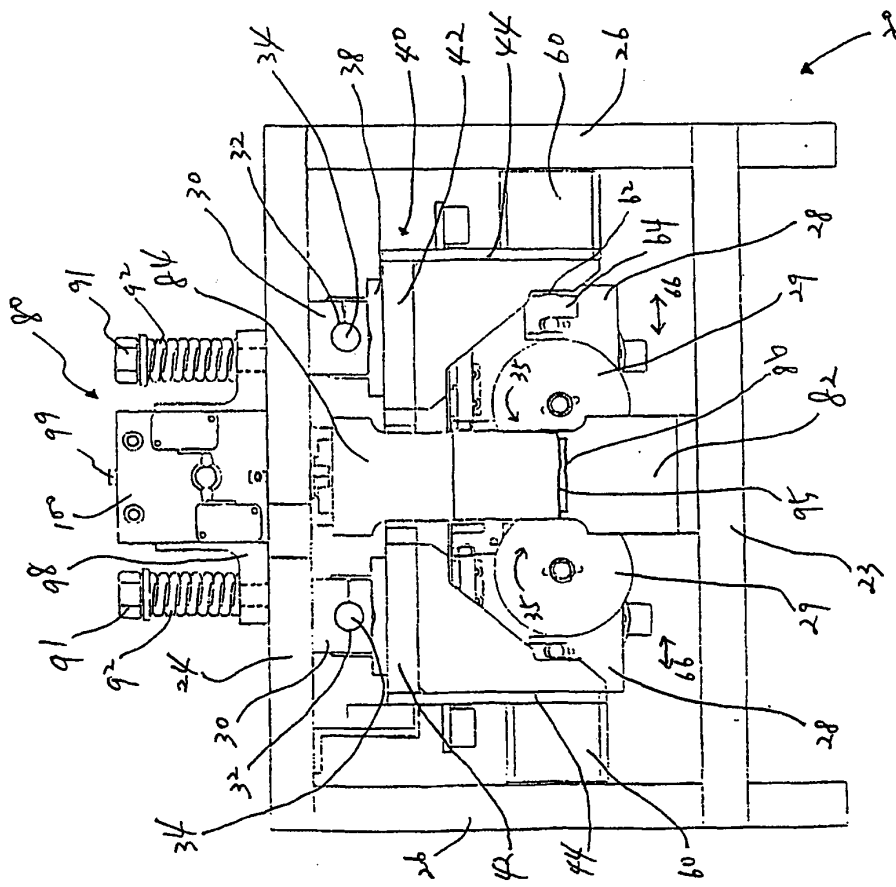
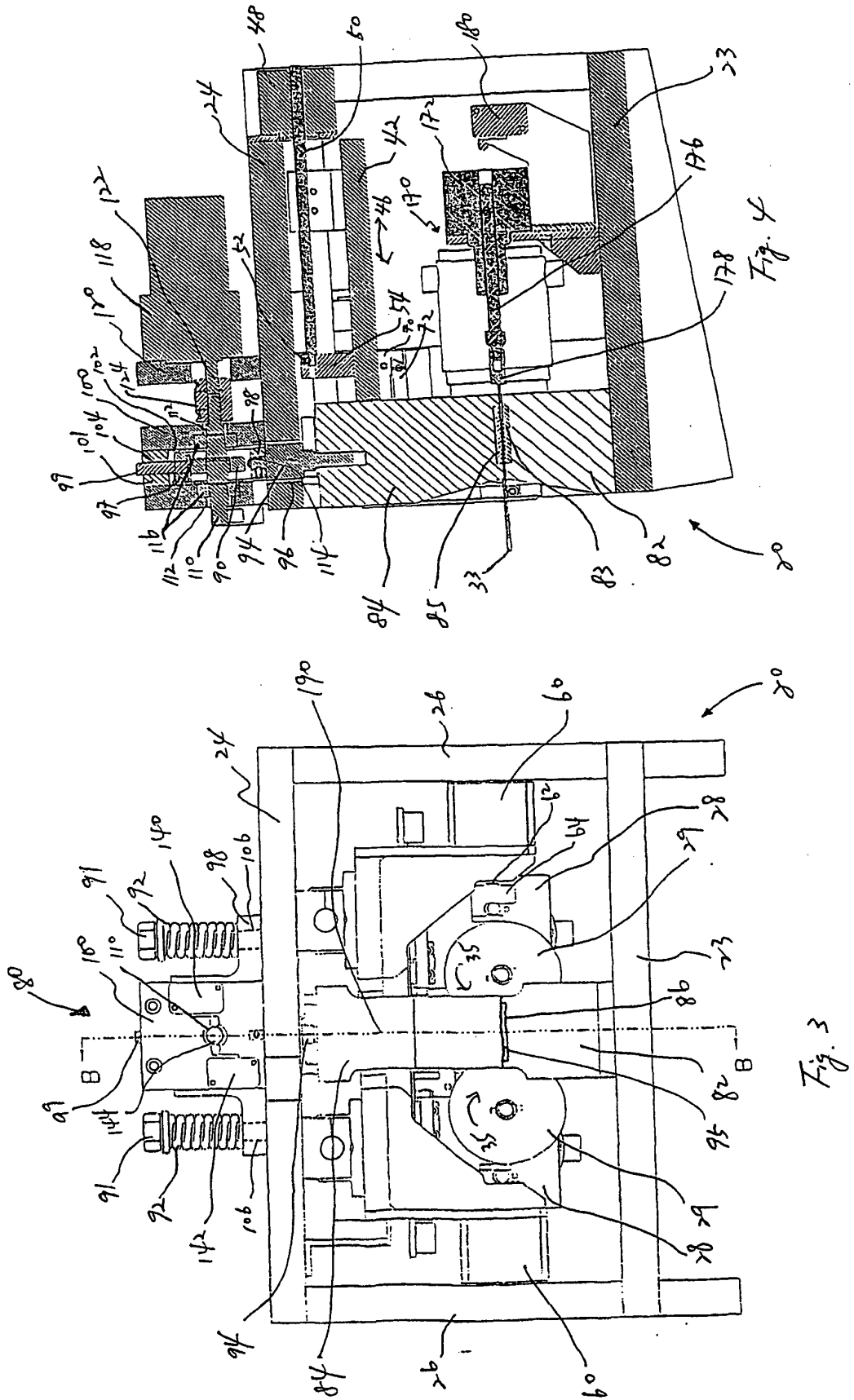
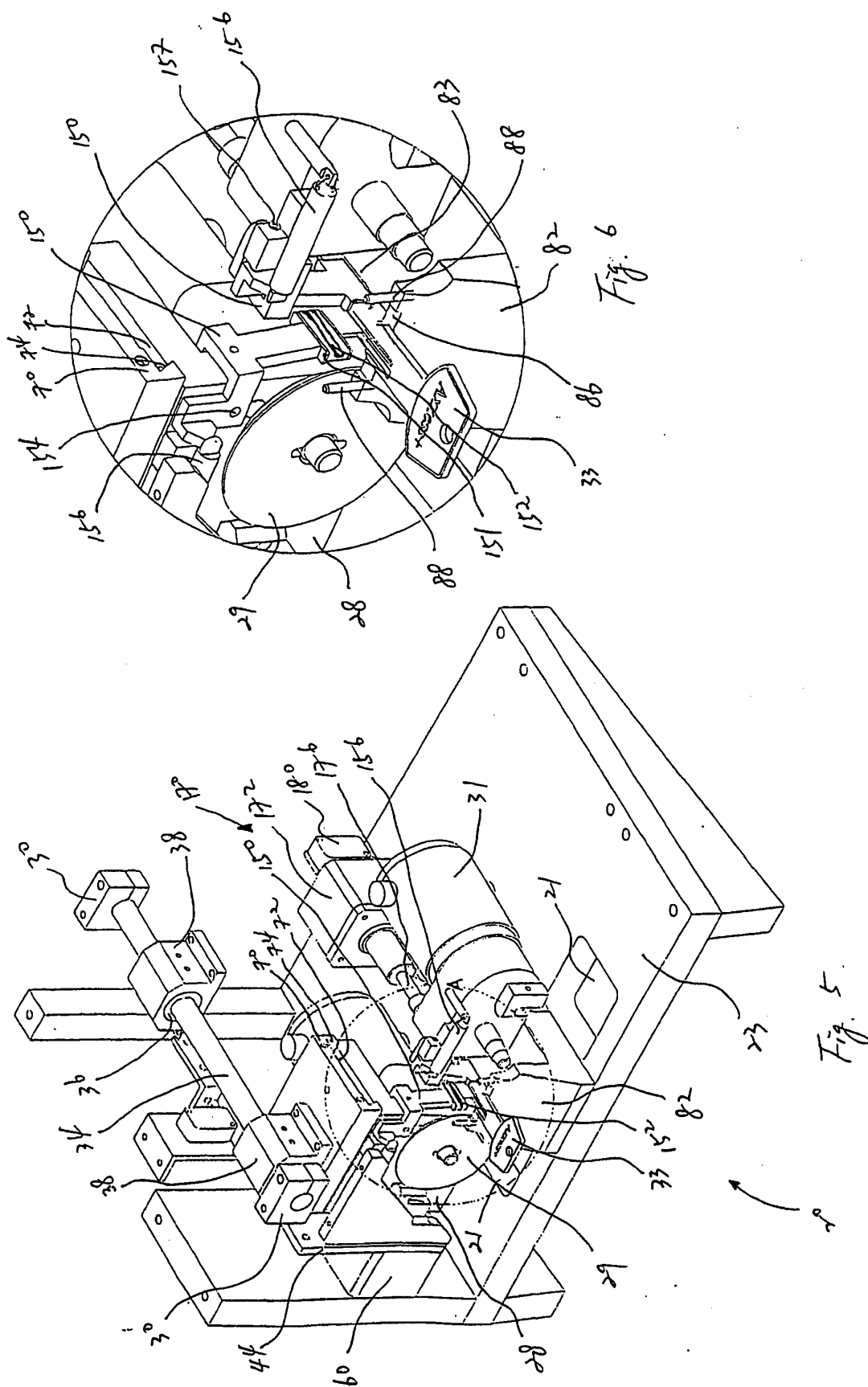


Fig. 2





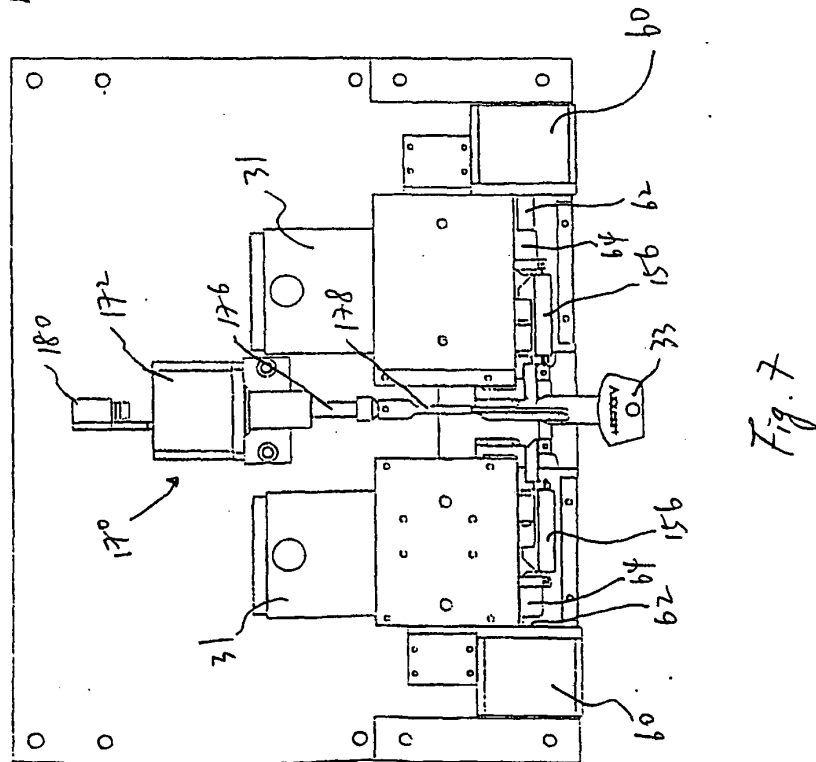
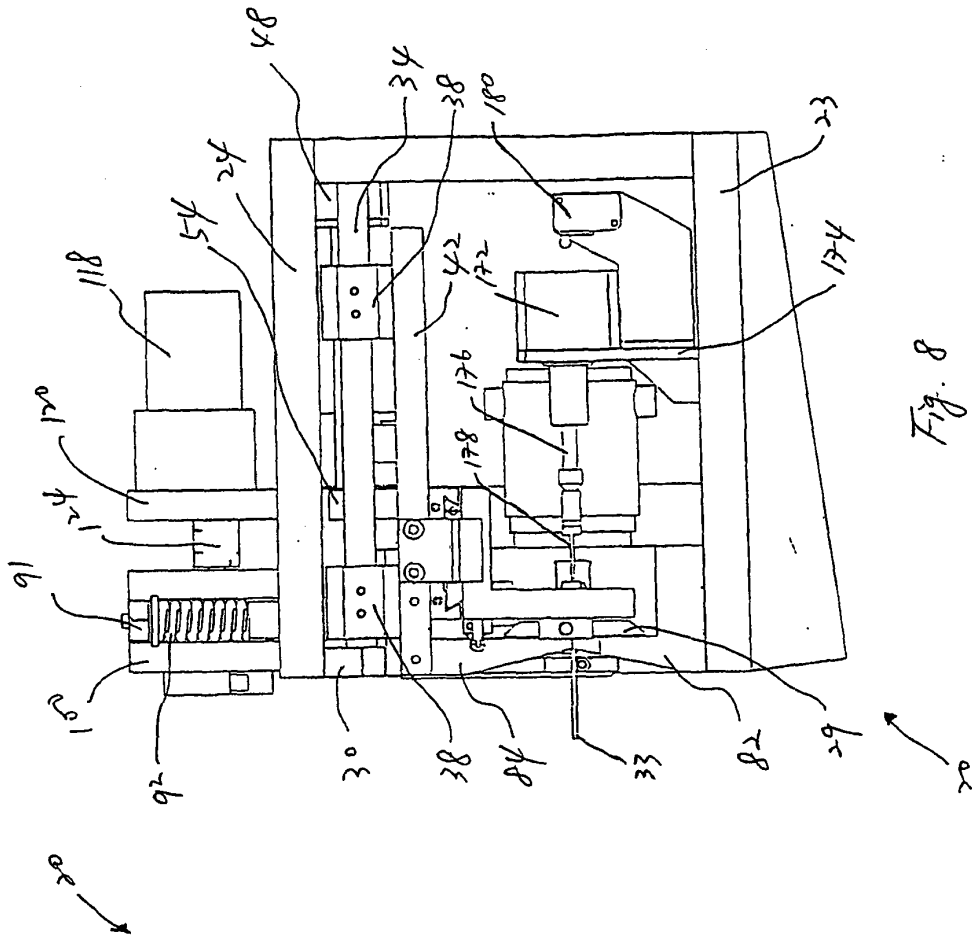


Fig. 9

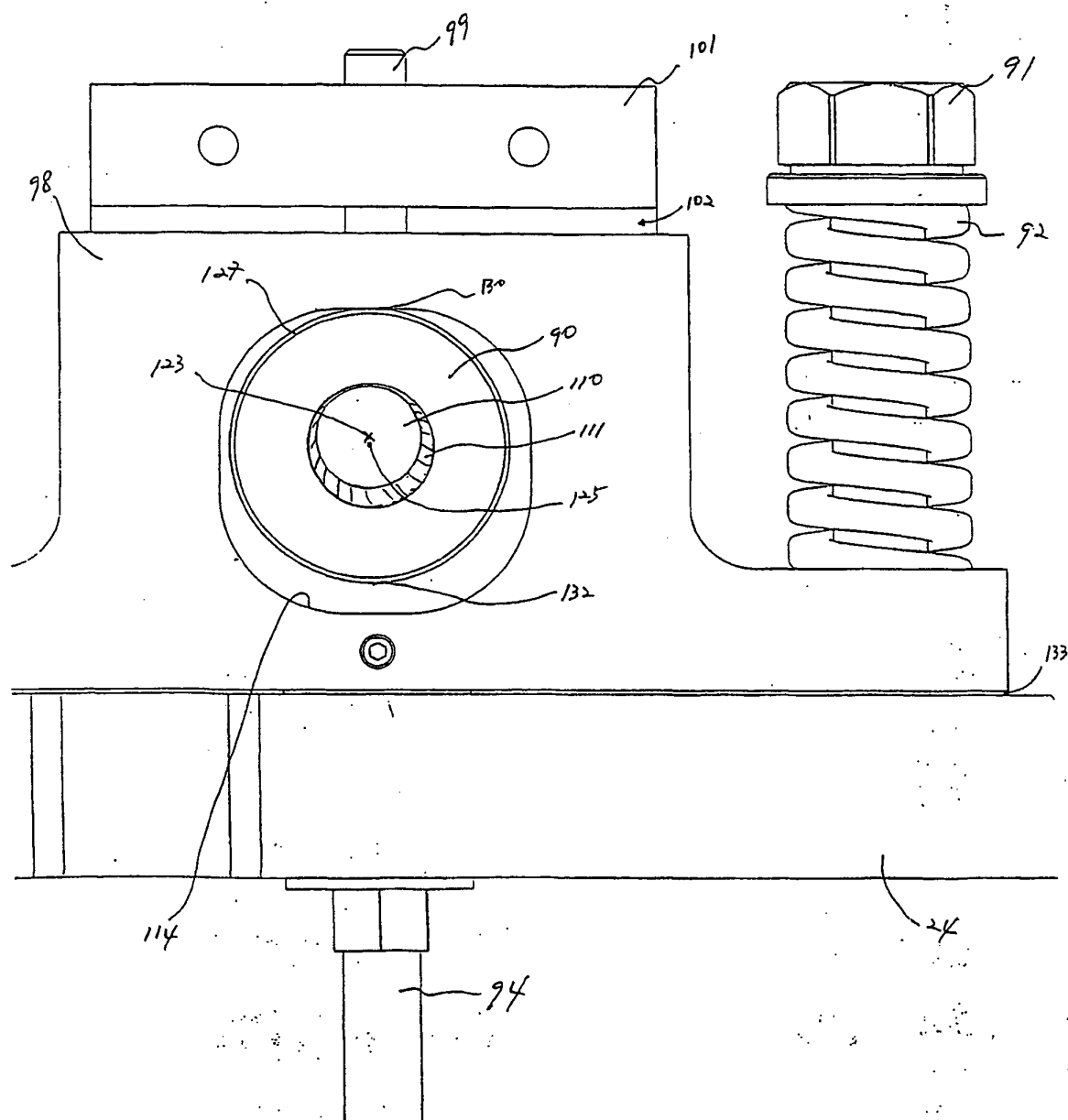
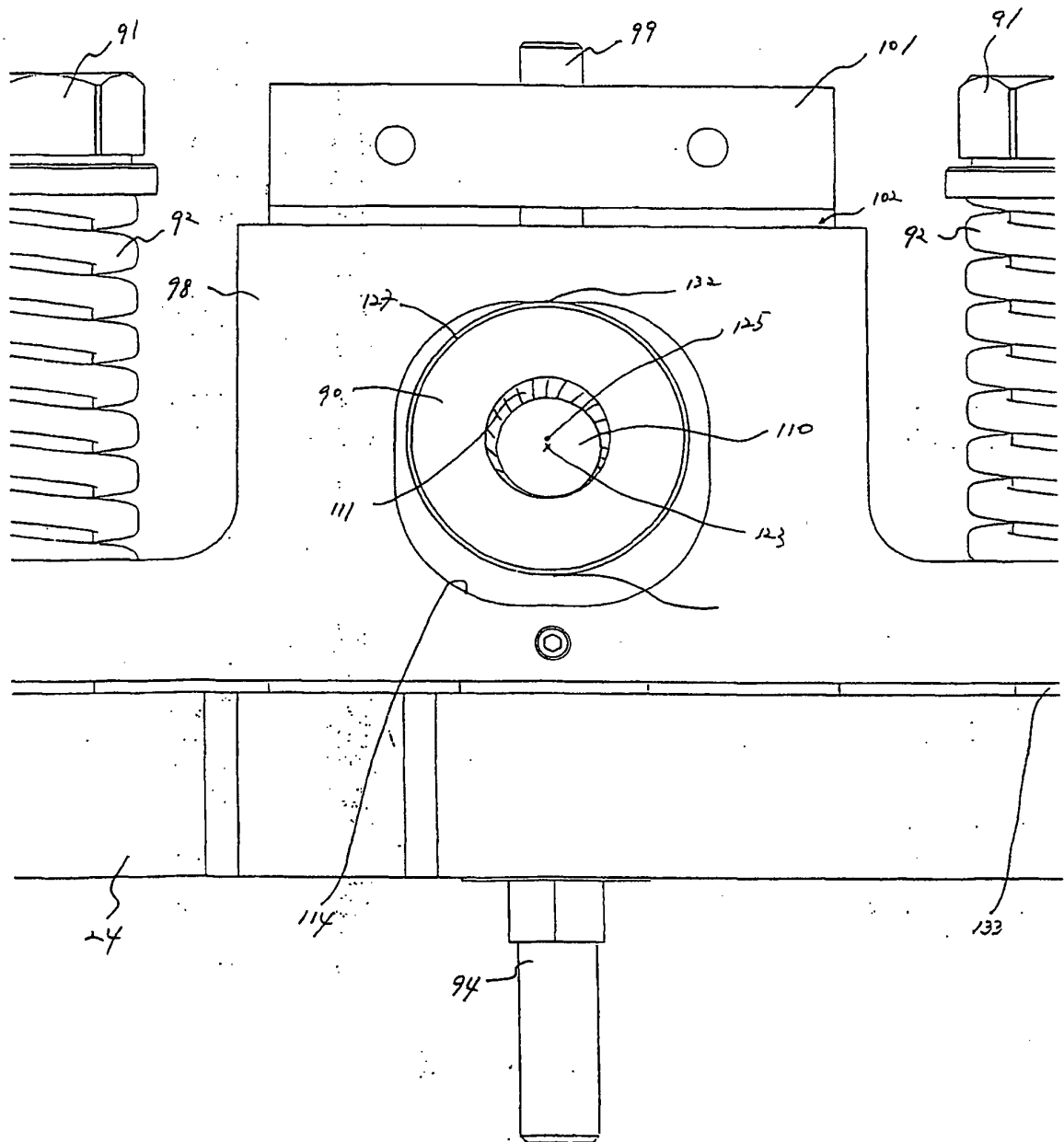


Fig. 10



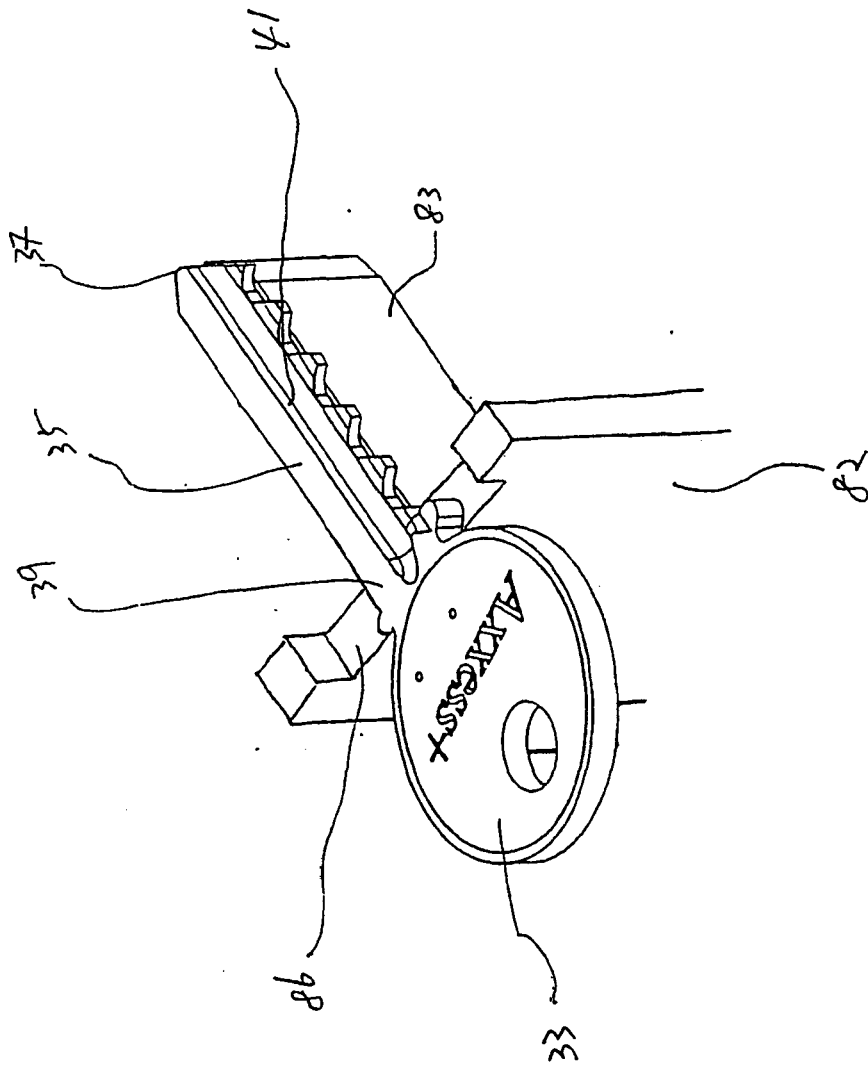
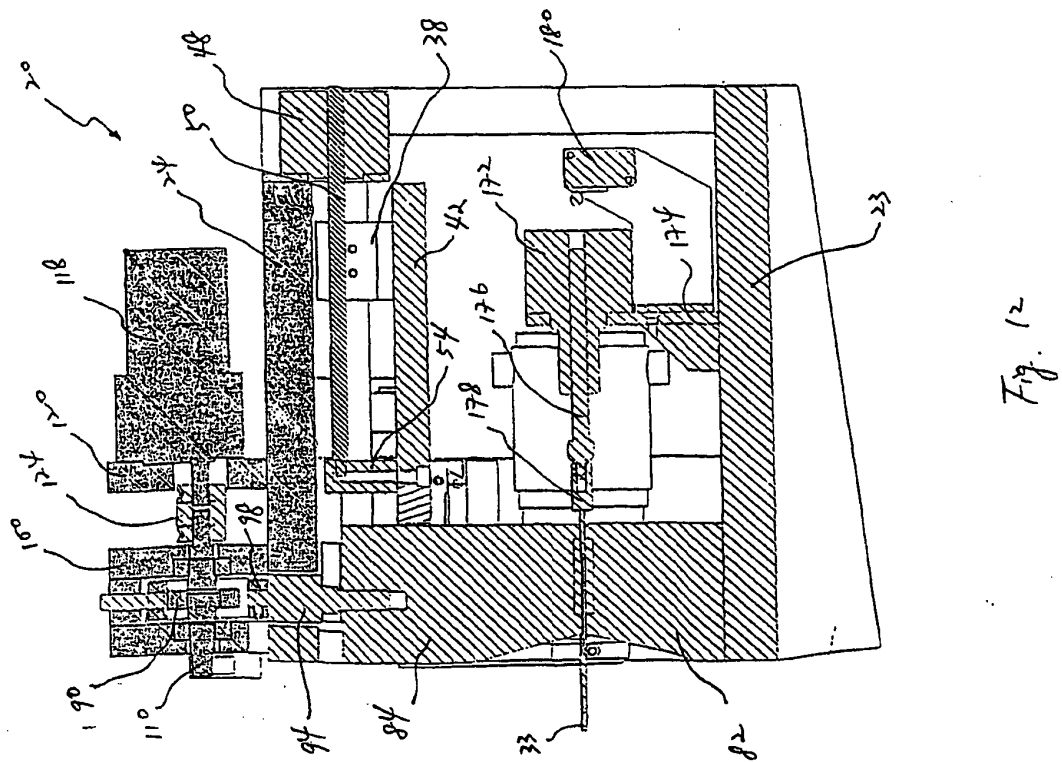
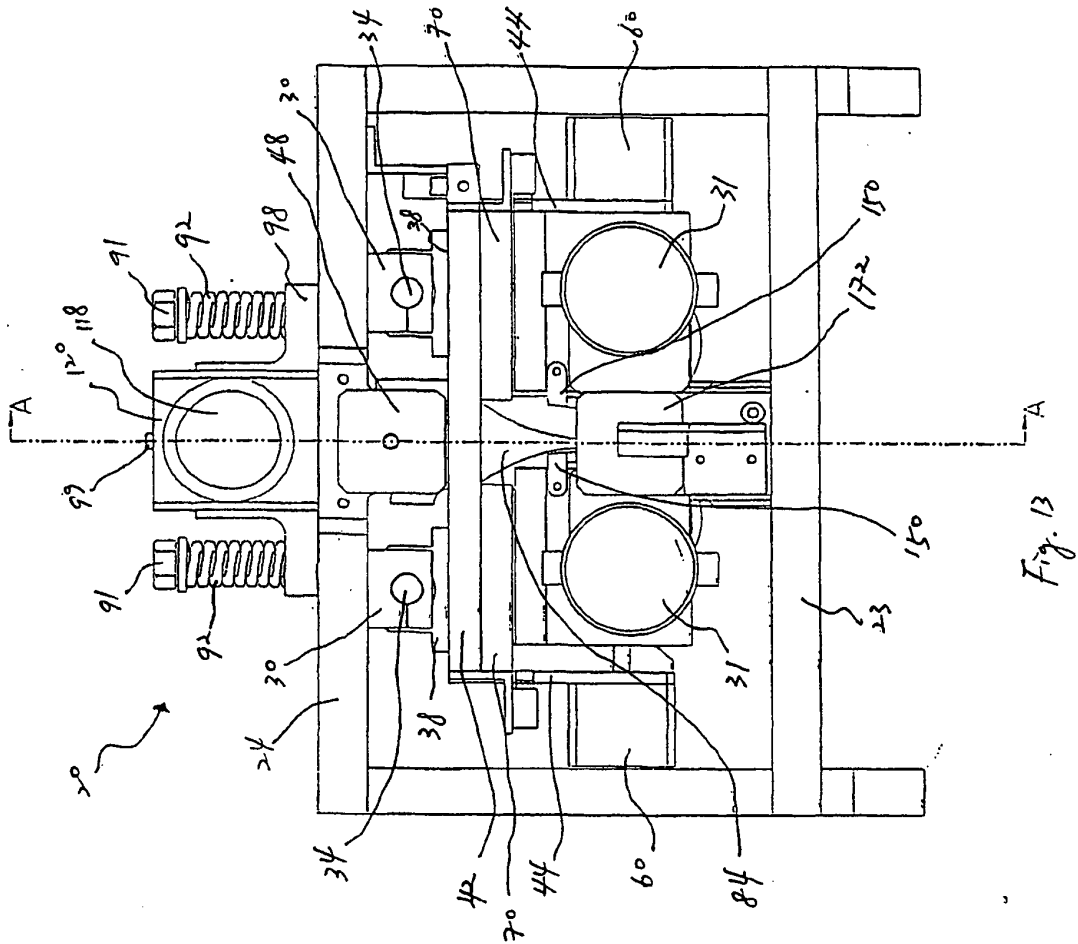


Fig. 11.



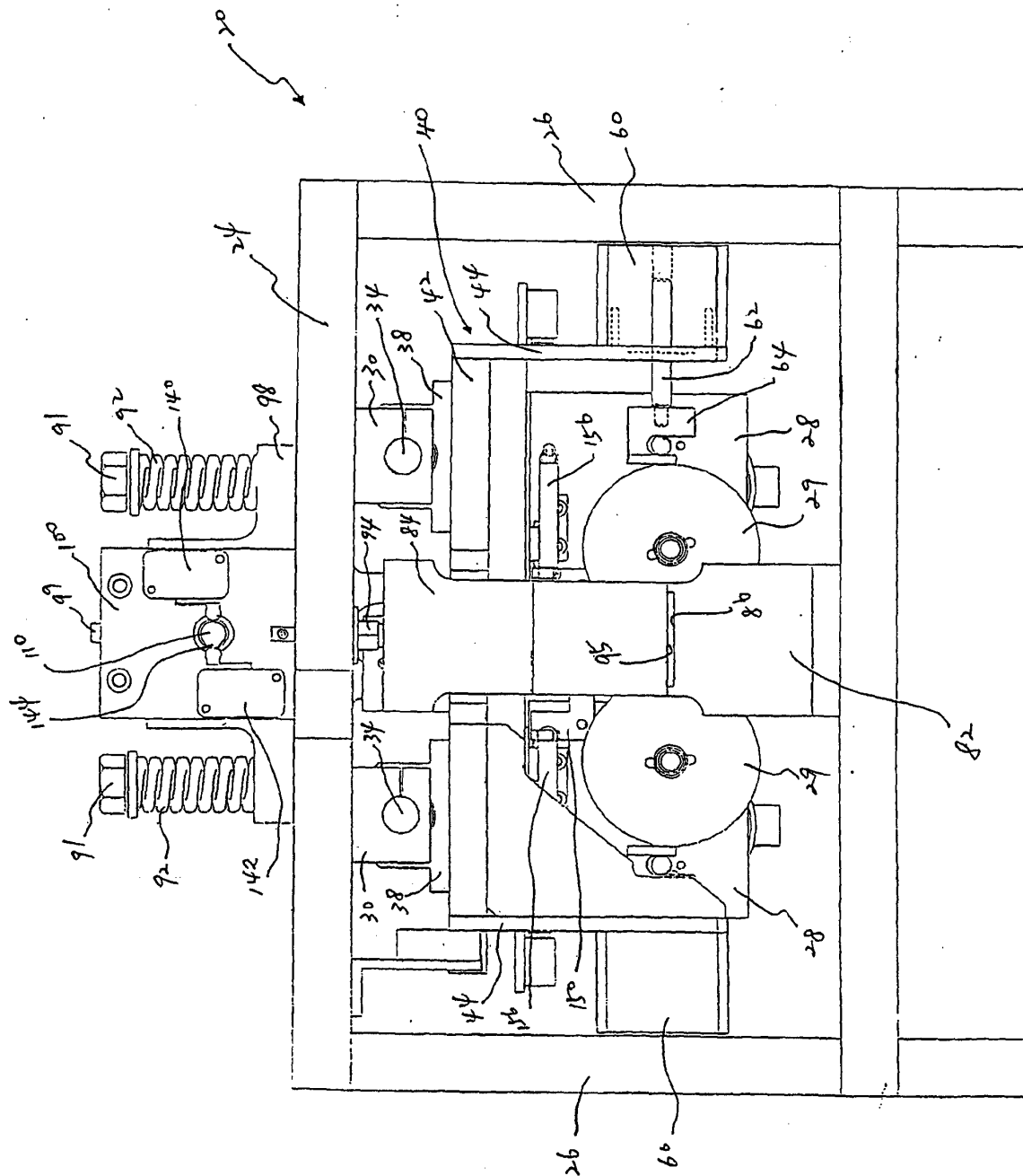


Fig. 14

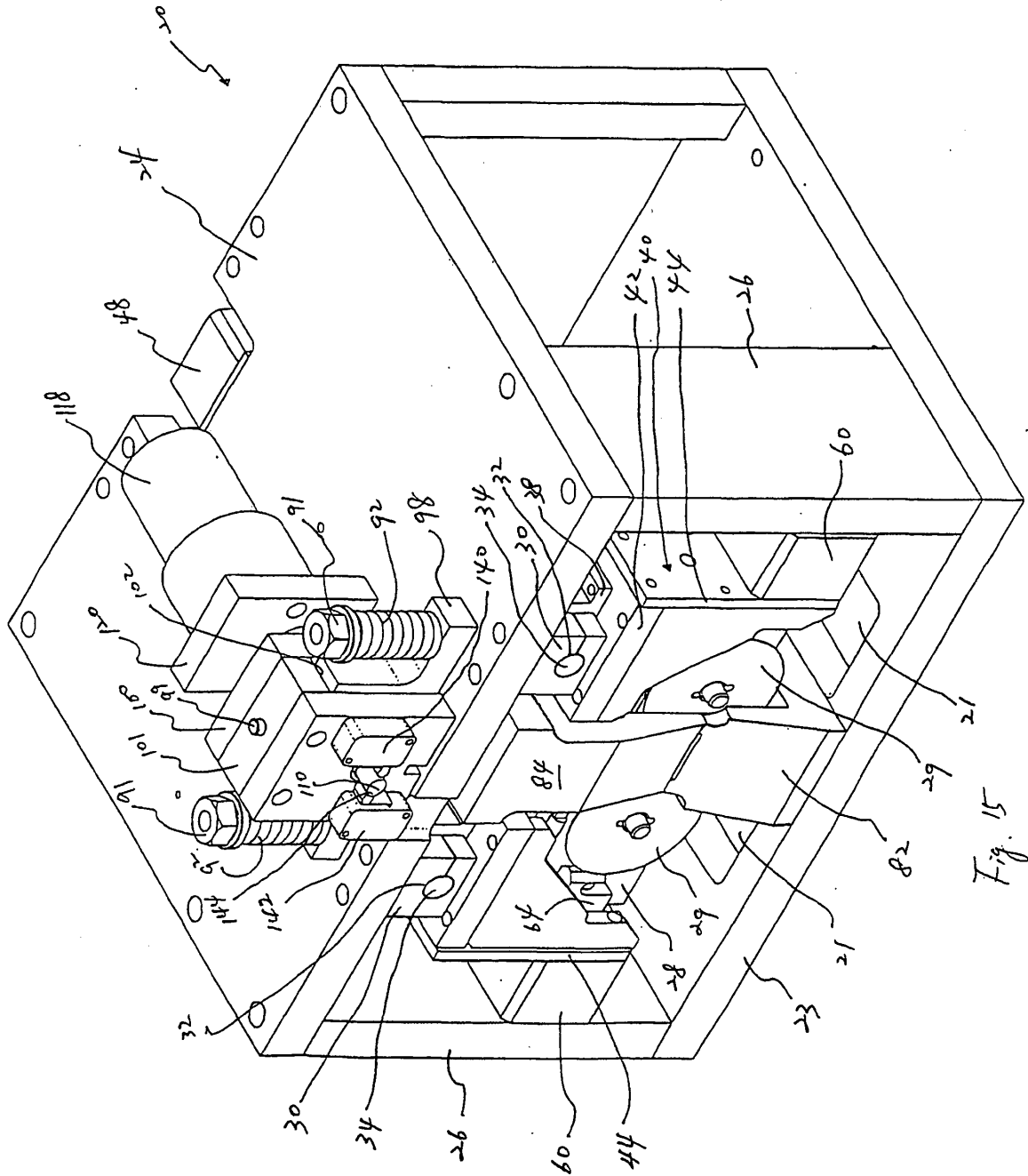


Fig. 15

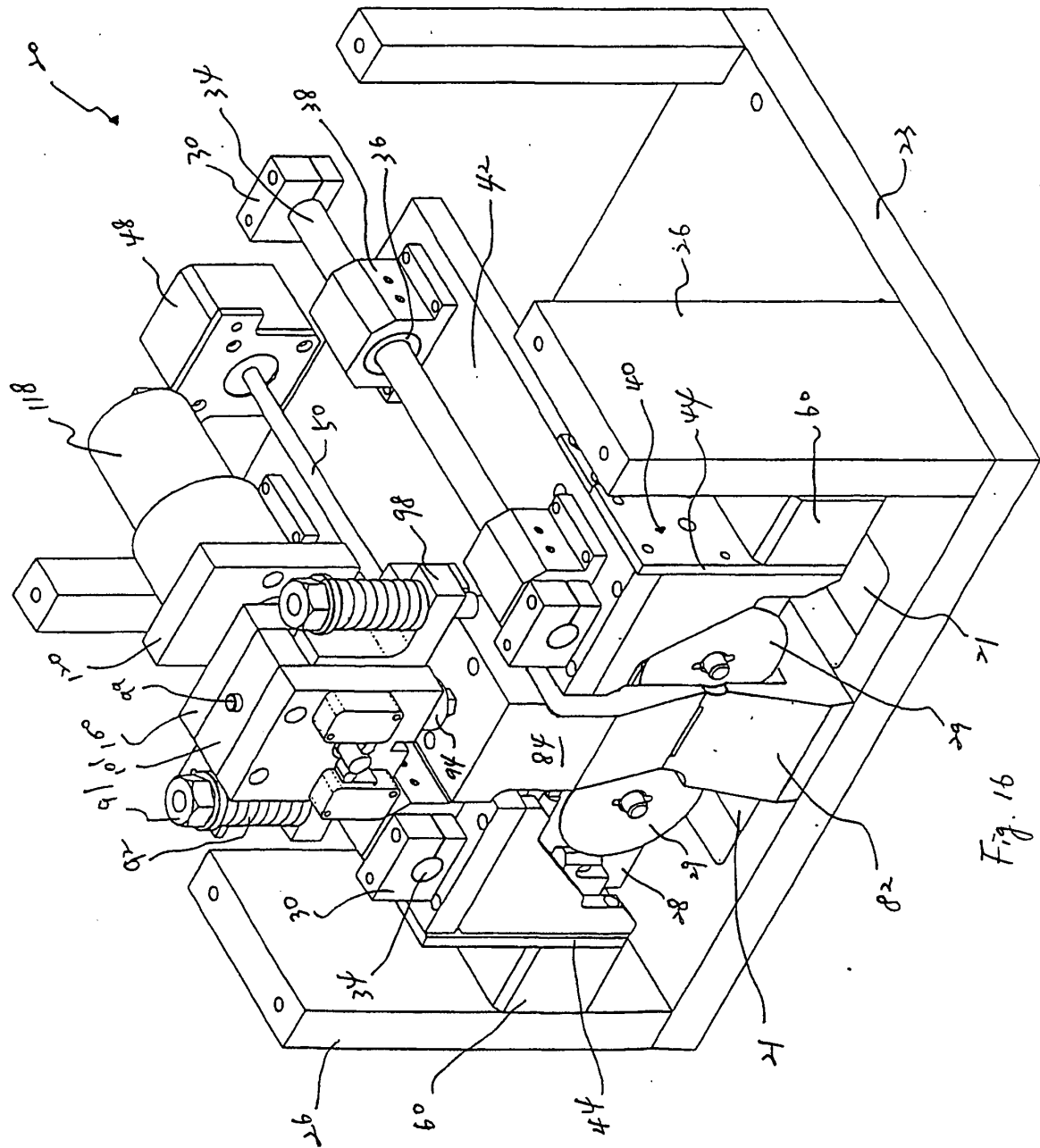
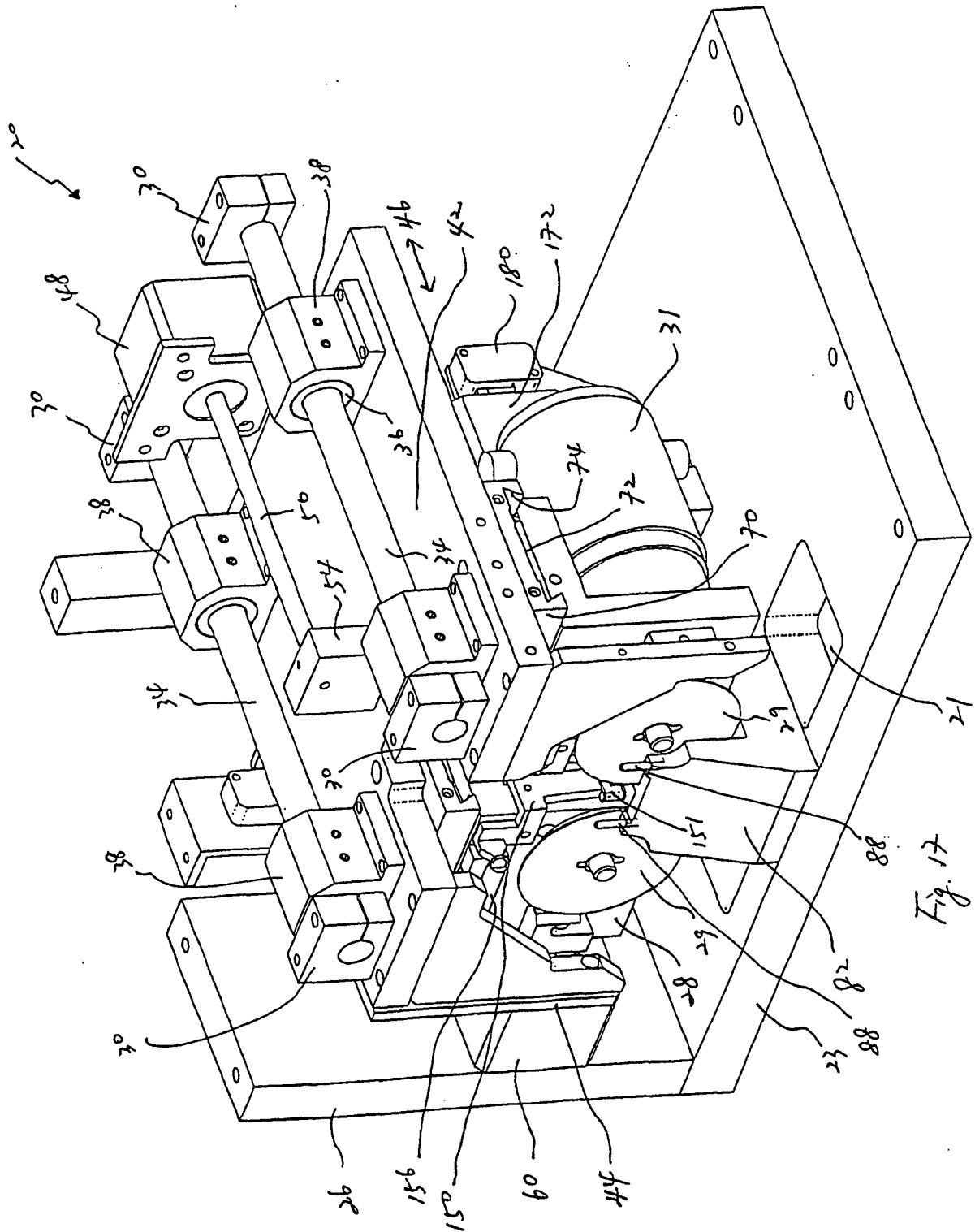
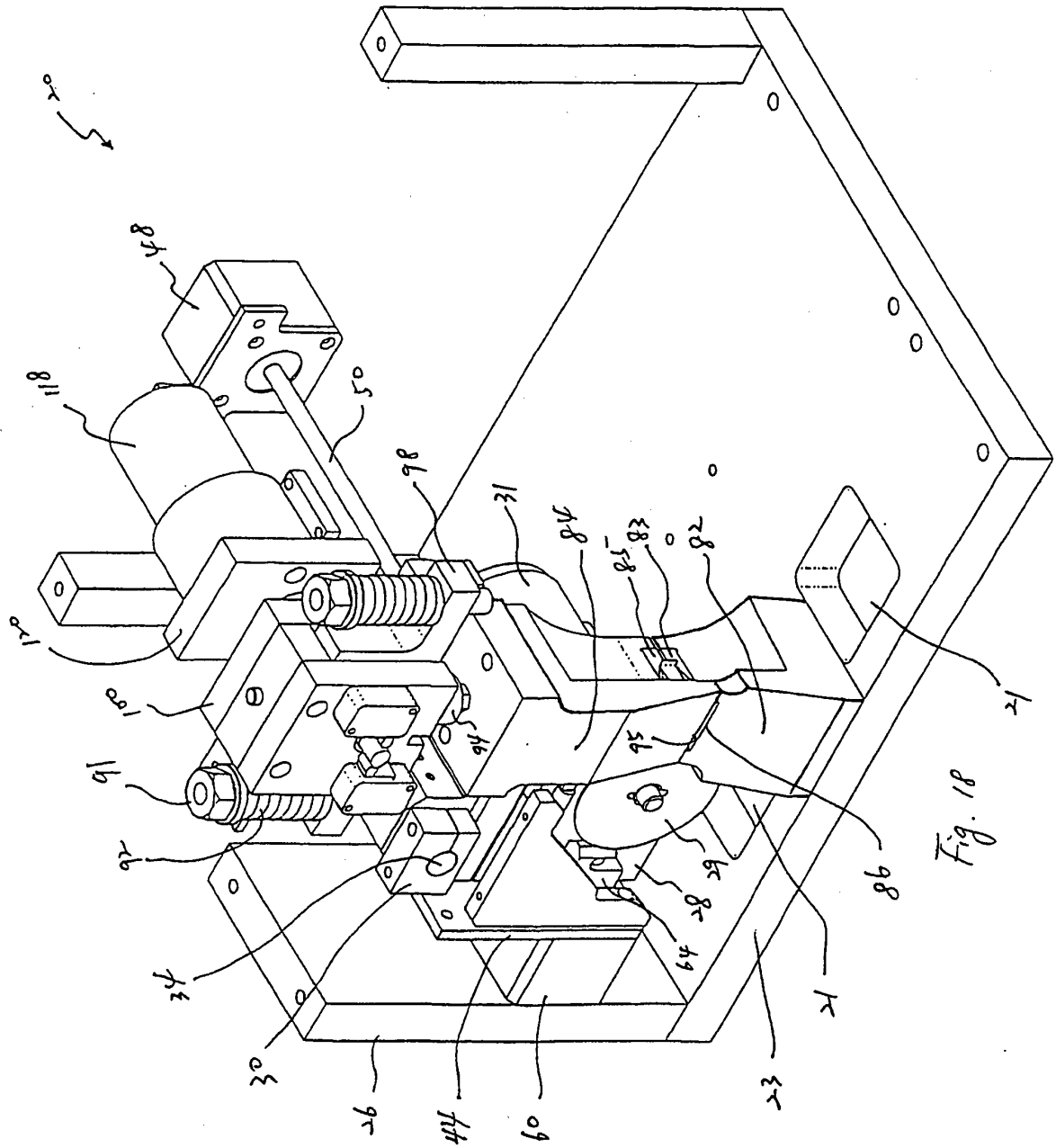


Fig. 16





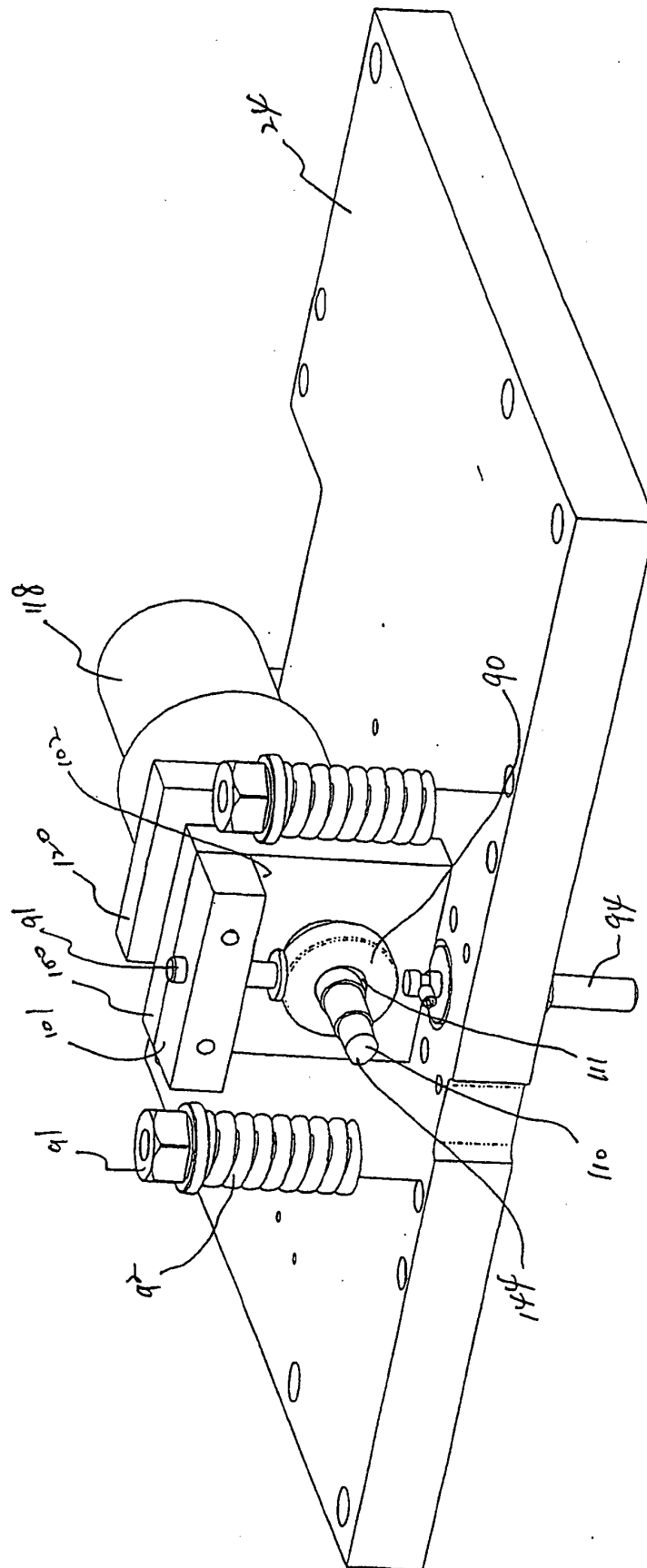
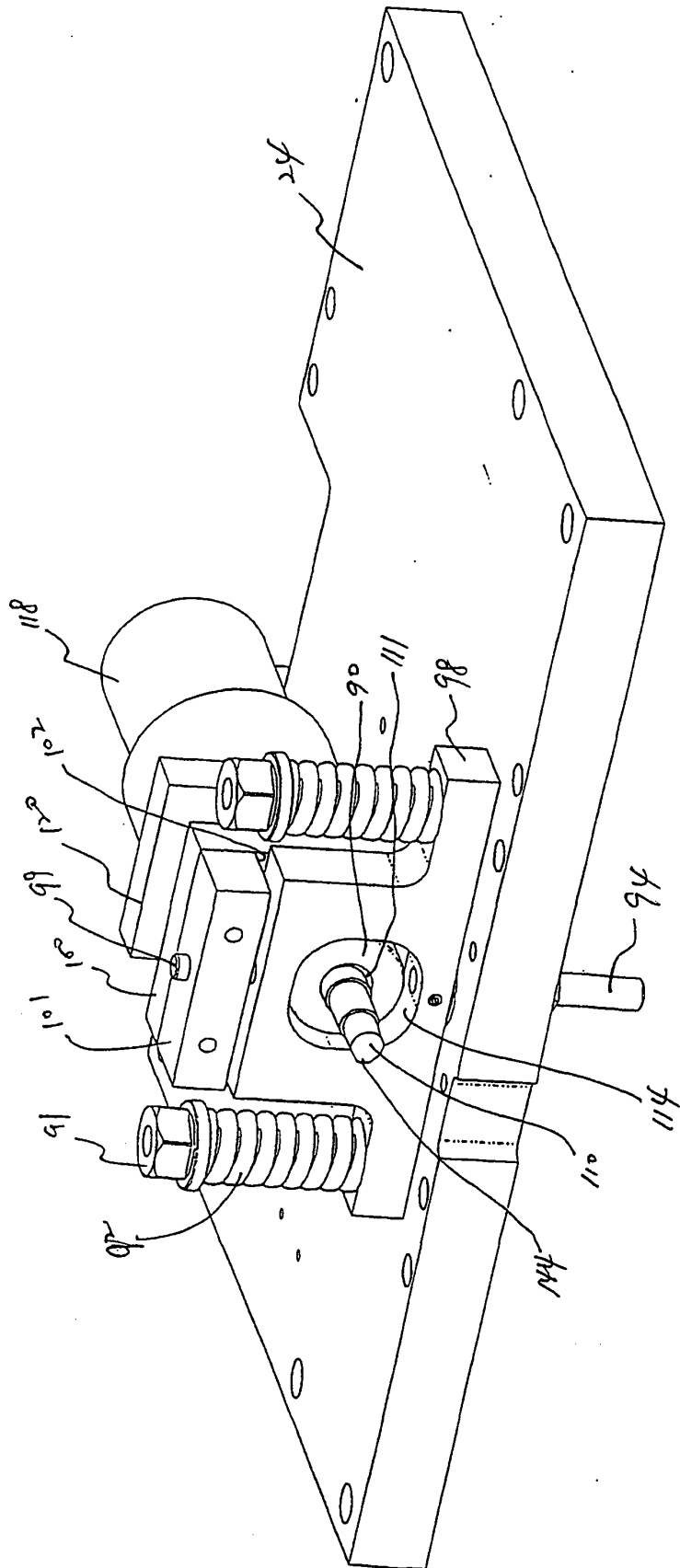


Fig. 19



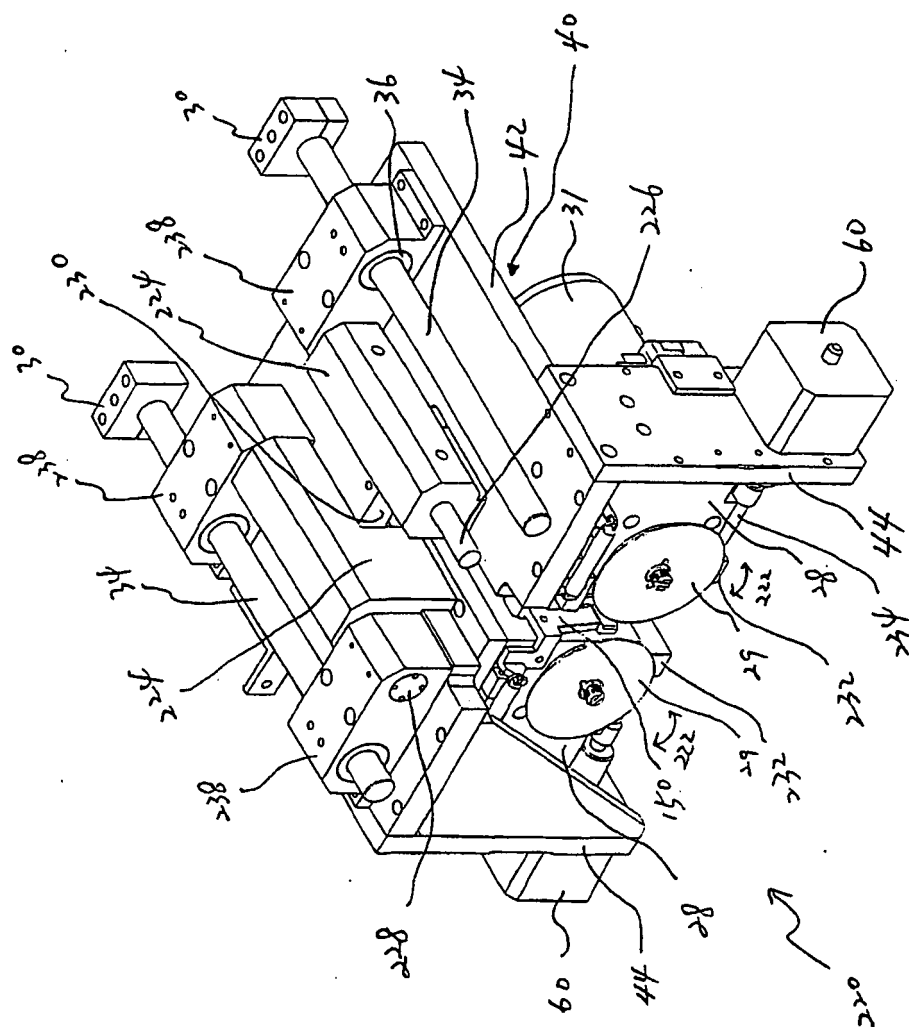


Fig. 21

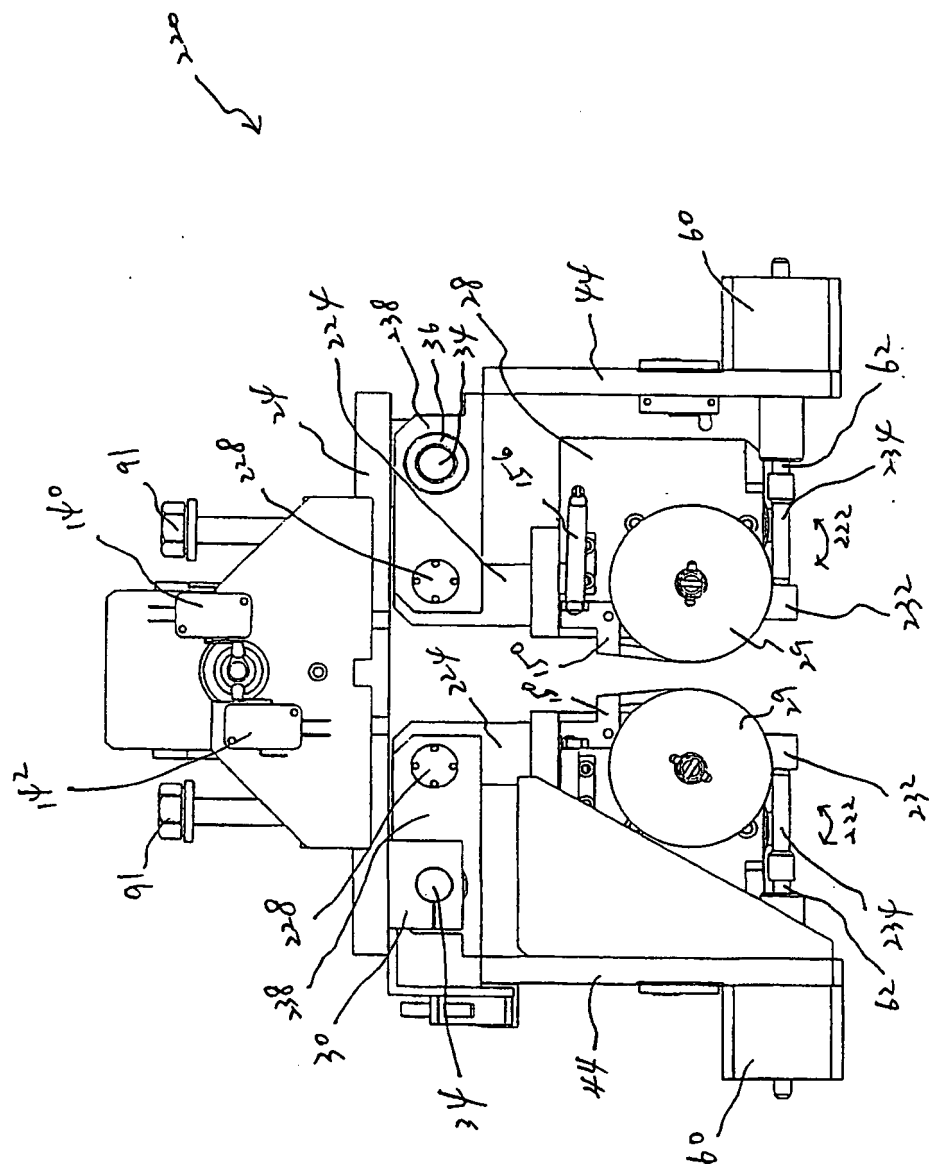


Fig. 22

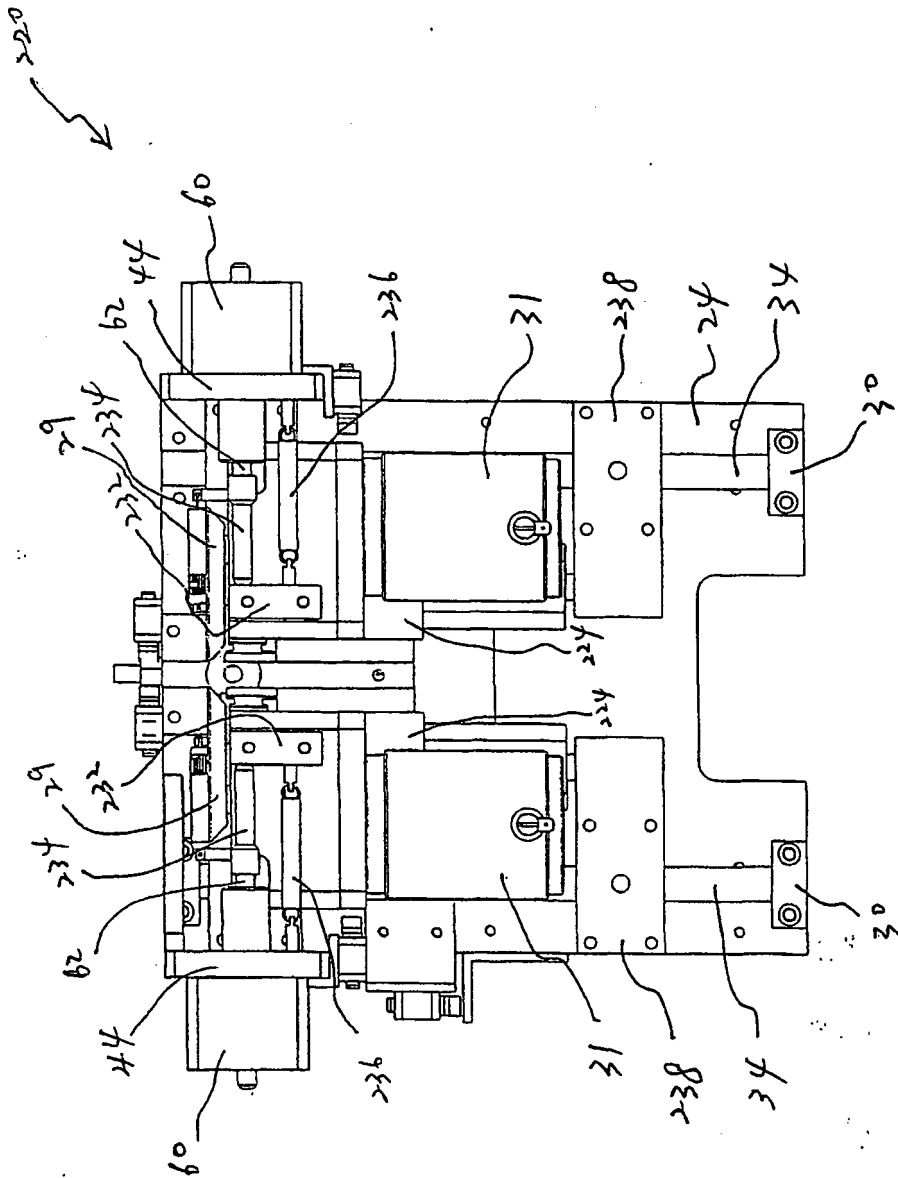


Fig. 23

REFERENCES CITED IN THE DESCRIPTION

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