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(54) **STABILIZER LEG ARRANGEMENT AND MOBILE WORKING MACHINE COMPRISING SUCH A STABILIZER LEG ARRANGEMENT**

STABILISATORSCHENKELANORDNUNG UND MOBILE ARBEITSMASCHINE MIT SOLCH EINER STABILISATORSCHENKELANORDNUNG

AGENCEMENT DE PIED STABILISATEUR ET MACHINE DE TRAVAIL MOBILE COMPORTANT UN TEL AGENCEMENT DE PIED STABILISATEUR

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Description

FIELD OF THE INVENTION AND PRIOR ART

[0001] The present invention relates to a stabilizer leg arrangement according to the preamble of claim 1 for supporting a mobile working machine against the ground. The invention also relates to a mobile working machine according to the preamble of claim 6.

[0002] A mobile working machine, for instance in the form of a vehicle having a hydraulic crane mounted on its chassis, is often provided with hydraulically actuated stabilizer legs for supporting the working machine against the ground to thereby improve the stability of the working machine. If such a working machine is provided with a system for monitoring the stability of the working machine, there might be a need to establish the magnitude of the support forces acting on the stabilizer legs when they are in supporting contact with the ground.

[0003] For a stabilizer leg provided with an actuator in the form of a hydraulic cylinder for extending the stabilizer leg vertically downwards into contact with the ground, the differential pressure in the hydraulic cylinder depends on the support force acting on the stabilizer leg when it is in supporting contact with the ground. The differential pressure in the hydraulic cylinder of such a stabilizer leg may be established based on measuring signals from a first pressure sensor configured to sense the hydraulic pressure on the piston side of the hydraulic cylinder and a from a second pressure sensor configured to sense the hydraulic pressure on the piston rod side of the hydraulic cylinder, and the magnitude of the support force acting on the stabilizer leg when it is in supporting contact with the ground may then be established based on the value of the differential pressure. However, a high differential pressure resembling the differential pressure caused by a support force on the stabilizer leg may also be developed in the hydraulic cylinder when it has reached its extended end position before the stabilizer leg makes contact with the ground. Thus, in order to avoid an erroneous detection of a support force on a stabilizer leg, a measuring system that is configured to use measuring values as to the differential pressure in the hydraulic cylinder of a stabilizer leg in order to establish the magnitude of the support force acting on the stabilizer leg also has to include an additional sensor for detecting whether or not the hydraulic cylinder is in its extended end position. Thus, a large number of sensors are required with this previously known solution.

[0004] It is also previously known to establish the magnitude of the support force acting on a stabilizer leg based on measuring values from a force sensor located in or close to a foot plate of the stabilizer leg. Such a solution is for instance disclosed in US 7 012 540 B2 and DE 102016104592 A1. A disadvantage with this type of solution is that the force sensor is located close to the ground when the stabilizer leg is in use, which implies an increased risk for damages to the force sensor. The ar-

rangement of a force sensor in or close to the foot plate also implies that the force sensor has to move together with the foot plate in both horizontal and vertical directions in connection with a movement of the stabilizer leg from its inactive resting position to its active operating position, which puts high demands on the cable connection between the force sensor and the electronic processing device that is configured to receive the measuring values from the force sensor.

[0005] JP H07 285786 A discloses an X-type outrigger arrangement. The hydraulic pressure in the pivoting cylinder is measured and, using this value, the ground reaction force is calculated.

[0006] JP H11 246176 A discloses a stabilizer leg arrangement according to the preamble of claim 1.

OBJECT OF THE INVENTION

[0007] The object of the present invention is to provide a new and favourable manner of establishing the magnitude of the support force acting on a stabilizer leg when supporting a mobile working machine against the ground.

SUMMARY OF THE INVENTION

[0008] According to the present invention, said object is achieved by means of a stabilizer leg arrangement having the features defined in claim 1.

[0009] The stabilizer leg arrangement according to the present invention comprises:

- a support beam to be fixedly connected to a base structure of a mobile working machine;
- an extension arm mounted to the support beam, the extension arm being telescopically extensible in order to allow an adjustment of the horizontal extension length thereof, wherein the extension arm comprises an extension beam which is telescopically mounted to the support beam so as to be axially slidable in relation to the support beam in the longitudinal direction of the extension arm in order to vary the horizontal extension length thereof, and wherein this extension beam extends into an interior space of the support beam through an entrance opening provided at an end of the support beam;
- a stabilizer leg mounted to the extension arm at an outer end thereof, the stabilizer leg being extensible in a vertical direction by means of an actuator, wherein a foot plate is arranged at a lower end of the stabilizer leg so as to be vertically moveable, by a vertical extension of the stabilizer leg under the effect of the actuator, from a raised position, in which the foot plate is lifted from the ground, to a lowered operating position, in which the foot plate is in supporting contact with the ground; and
- a measuring system for establishing the magnitude of the support force acting on said foot plate in the operating position, wherein the measuring system

comprises:

- a length measuring device configured to generate a first measuring value that depends on the extension length of the extension arm;
- a force sensor which is arranged on the support beam in the vicinity of the entrance opening and which is configured to generate a second measuring value that depends on the contact force between said extension beam and the support beam at the entrance opening; and
- an electronic processing device configured to establish the magnitude of the support force based on said first and second measuring values.

[0010] With a measuring system of the above-mentioned type, the magnitude of the support force acting on the stabilizer leg may be established in a simple manner by means of a low number of sensors and without requiring any use of a sensor mounted to a moveable part of the extension arm or to the stabilizer leg. On the contrary, all the required sensors may be fixed to the support beam, which implies a reduced risk for damages to the sensors and makes it possible for the sensors to be connected to the electronic processing device in a simple and reliable manner.

[0011] Further advantageous features of the stabilizer leg arrangement according to the present invention will appear from the description following below and the dependent claims.

[0012] The invention also relates to a mobile working machine having the features defined in claim 6.

[0013] Further advantageous features of the mobile working machine according to the present invention will appear from the description following below and the dependent claims.

BRIEF DESCRIPTION OF THE DRAWINGS

[0014] The invention will in the following be more closely described by means of embodiment examples, with reference to the appended drawings. In the drawings:

Fig 1 is a schematic perspective view of a mobile working machine provided with a hydraulic crane and a stabilizer leg arrangement according to an embodiment of the present invention,

Fig 2 is a partly cut outline diagram of parts included in a stabilizer leg arrangement according to an embodiment of the invention, with the stabilizer leg shown in a raised non-use position,

Fig 3 is a partly cut outline diagram corresponding to Fig 2, with the stabilizer leg shown in a lowered operating position,

Fig 4 is a schematic section according to the line IV-IV in Fig 2, and

Fig 5 is an outline diagram of a measuring system included in a stabilizer leg arrangement according to the invention.

DETAILED DESCRIPTION OF EMBODIMENTS OF THE INVENTION

[0015] Fig 1 schematically illustrates a mobile working machine 1 provided with a stabilizer leg arrangement 2 according to an embodiment of the present invention, wherein the stabilizer leg arrangement 2 is to be used for supporting the mobile working machine against the ground. In the illustrated example, the mobile working machine 1 is a crane vehicle comprising a vehicle 3 and a hydraulic crane 4. The crane 4 is mounted to a base structure 5 of the mobile working machine 1. In the illustrated example, said base structure 5 constitutes the chassis of the vehicle 3.

[0016] The illustrated crane 4 comprises:

- a crane base 6, which is fixed to the chassis of the vehicle 3;
- a crane column 7 which is rotatably mounted to the crane base 6 so as to be rotatable in relation to the crane base about an essentially vertical axis of rotation;
- a liftable and lowerable first crane boom 8, which is articulately connected to the crane column 7 in such a manner that it is pivotable in relation to the crane column about an essentially horizontal axis of rotation;
- a first hydraulic cylinder 9 for lifting and lowering the first crane boom 8 in relation to the crane column 7;
- a liftable and lowerable second crane boom 10, which is articulately connected to the first crane boom 8 in such a manner that it is pivotable in relation to the first crane boom 8 about an essentially horizontal axis of rotation; and
- a second hydraulic cylinder 11 for lifting and lowering the second crane boom 10 in relation to the first crane boom 8.

[0017] The above-mentioned second crane boom 10 is telescopically extensible by means of a number of hydraulic cylinders 12 in order to enable an adjustment of the extension length thereof.

[0018] The stabilizer leg arrangement 2 comprises a support beam 20 fixedly connected to a base structure 5 of the mobile working machine 1, i.e. to the chassis of the vehicle 3 included in the mobile working machine 1. In the embodiment illustrated in Fig 1, the support beam 20 is rigidly mounted to the crane base 6, wherein the support beam 20 is fixedly connected to the base structure 5 of the mobile working machine 1 via the crane base 6. However, the support beam 20 may as an alternative

be mounted directly to the base structure 5 of the mobile working machine 1.

[0019] In embodiment illustrated in Fig 1, two extension arms 21 are mounted to the support beam 20 on opposite sides thereof. Each extension arm 21 is telescopically extensible in order to allow an adjustment of the horizontal extension length thereof. Each extension arm 21 comprises at least a first extension beam 22 which is telescopically mounted to the support beam 20 so as to be axially slidable in relation to the support beam 20 in the longitudinal direction of the extension arm 21 in order to vary the horizontal extension length thereof. This first extension beam 22 extends into an interior space 23 (see Figs 2 and 3) of the support beam 20 through an entrance opening 24 provided at an end of the support beam. A reinforcement collar 25, which forms part of the support beam 20, is arranged around each entrance opening 24. The reinforcement collar 25 extends around the rim of the associated entrance opening 24 and is arranged in a plane perpendicular to the longitudinal axis of the support beam 20. The first extension beam 22 is horizontally moveable in relation to the support beam 20 by means of a hydraulic cylinder (not shown) or any other suitable type of linear actuator. Each extension arm 21 may also comprise a further extension beam which is telescopically mounted to the first extension beam 22 so as to be axially slidable in relation to the first extension beam. However, in the illustrated embodiment, each extension arm 21 comprises only one extension beam 22.

[0020] A stabilizer leg 26 is mounted to each extension arm 21 at an outer end thereof, wherein the stabilizer leg 26 is extensible in a vertical direction by means of an actuator 27. Each stabilizer leg 26 comprises a foot plate 28, which is arranged at a lower end of the stabilizer leg so as to be vertically moveable, by a vertical extension of the stabilizer leg 26 under the effect of the actuator 27, from a raised non-use position (see Fig 2), in which the foot plate 28 is lifted from the ground 29, to a lowered operating position (see Fig 3), in which the foot plate 28 is in supporting contact with the ground 29.

[0021] In the illustrated embodiment, the actuator 27 of each stabilizer leg 26 has the form of a hydraulic cylinder with a cylinder part 27a, which forms an upper part of the stabilizer leg, and a piston rod 27b, which forms a lower part of the stabilizer leg. The piston rod 27b is at its upper end fixed to a piston (not shown), which is received in the cylinder part 27a and displaceable in relation to it. The foot plate 28 is fixed to the lower end of the piston rod 27b.

[0022] The external cross-sectional shape of the first extension beam 22 is somewhat smaller than the internal cross-sectional shape of the entrance opening 24 and the internal cross-sectional shape of the interior space 23 of the support beam 20.

[0023] When the foot plate 28 of a stabilizer leg 26 is in the raised non-use position illustrated in Fig 2, the first extension beam 22 is slightly inclined downwards as seen from the rear end 22a of the first extension beam

towards the front end thereof, wherein the bottom surface 30a of the first extension beam 22 abuts against the lower edge 31a of the entrance opening 24 and the upper surface 30b of the first extension beam 22 abuts against the upper surface 32b of the interior space 23 of the support beam 20 at the rear end 22a of the first extension beam 22. There is consequently, in this situation, a small gap between the upper surface 30b of the first extension beam 22 and the upper edge 31b of the entrance opening 24 as well as between the bottom surface 30a of first the extension beam 22 and the bottom surface 32a of the interior space 23 of the support beam 20.

[0024] When the foot plate 28 of a stabilizer leg 26 is in the lowered operating position illustrated in Fig 3, the first extension beam 22 is slightly inclined upwards as seen from the rear end 22a of the first extension beam 22 towards the front end thereof, wherein the upper surface 30b of the first extension beam 22 abuts against the upper edge 31b of the entrance opening 24 and the bottom surface 30a of the first extension beam 22 abuts against the bottom surface 32a of the interior space 23 of the support beam 20 at the rear end 22a of the first extension beam 22. There is consequently, in this situation, a small gap between the bottom surface 30a of the first extension beam 22 and the lower edge 31a of the entrance opening 24 as well as between the upper surface 30b of the first extension beam 22 and the upper surface 32b of the interior space 23 of the support beam 20.

[0025] The stabilizer leg arrangement 2 further comprises a measuring system 40 (very schematically illustrated in Fig 5) for establishing the magnitude of the support force F3 acting on the foot plate 28 of each stabilizer leg 26 when the foot plate is in the operating position in contact with the ground 29. The support force F3 corresponds to the contact force between the ground 29 and the foot plate 28. The measuring system 40 comprises a length measuring device 41 configured to generate a first measuring value that depends on the extension length of the extension arm 21. Thus, the length measuring device 41 is configured to emit a measuring signal that varies in dependence on the extension length of the extension arm 21. The length measuring device 41 is preferably of a type which utilizes a noncontact measurement technique. The length measuring device 41 is for instance of magneto-resistive type or laser type. However, an electromechanical length measuring device or a length measuring device of any other suitable type may also be used.

[0026] The measuring system 40 also comprises a force sensor 42 which is arranged on the support beam 20 in the vicinity of the entrance opening 24 and which is configured to generate a second measuring value that depends on the contact force F2 between the first extension beam 22 and the support beam 20 at the entrance opening 24. Thus, this force sensor 42 is configured to emit a measuring signal that varies in dependence on the contact force F2. The force sensor 42 preferably com-

prises one or more strain gauges configured to sense the strain on a part of the support beam 20 in the vicinity of the entrance opening 24 caused by the contact force F2. The strain on the support beam 20 in the vicinity of the entrance opening 24 has a magnitude that depends on the magnitude of the contact force F2, and the magnitude of the contact force F2 can thereby be derived from the magnitude of said strain. The force sensor 42 may as an alternative be a piezoresistive force sensor or a force sensor of any other suitable type. The force sensor 42 is with advantage arranged on the reinforcement collar 25, as illustrated in Fig 4, but it may as an alternative be arranged on a lateral wall of the support beam 20 close to the entrance opening 24.

[0027] The length measuring device 41 and the force sensor 42 are connected to an electronic processing device 43, which is configured to receive the above-mentioned first and second measuring values from the length measuring device 41 and the force sensor 42.

[0028] The contact forces acting on the foot plate 28 and on the first extension beam 22 when the foot plate 28 is in the lowered operating position are illustrated in Fig 3. The foot plate 28 is subjected to an upwardly directed contact force at the interface between the foot plate and the ground 29, i.e. the support force F3, whereas the first extension beam 22 is subjected to a downwardly directed contact force F2 at the interface between the first extension beam and the upper edge 31b of the entrance opening 24 and an upwardly directed contact force F1 at the interface between the first extension beam 22 and the bottom surface 32a of the interior space 23 of the support beam 20. Equilibrium of moments about the point of contact between the first extension beam 22 and the support beam 20 at the entrance opening 24 will give the following relationship:

$$F1 \cdot L1 - F3 \cdot L2 = 0 \quad [1]$$

where:

- F1 is the contact force between the first extension beam 22 and the support beam 20 at the rear end 22a of the first extension beam,
- F3 is the support force,
- L2 is the horizontal distance between the entrance opening 24 and the centre axis of the stabilizer leg 26, which essentially corresponds to the extension length of the extension arm 21, and
- L1 is the horizontal distance between the entrance opening 24 and the rear end 22a of the first extension beam 22.

[0029] Also the following relationships can be concluded from Fig 3:

$$F1 - F2 + F3 = 0 \quad [2]$$

$$L1 = L - L2 \quad [3]$$

where F2 is the contact force between the first extension beam 22 and the support beam 20 at the entrance opening 24 and L is the horizontal distance between the rear end 22a of the first extension beam 22 and the centre axis of the stabilizer leg 26. In the illustrated example, the distance L essentially corresponds to the length of the first extension beam 22.

[0030] From the above-mentioned relationships [1], [2] and [3] it follows that:

$$F3 = \frac{F2 \cdot (L - L2)}{L} \quad [4]$$

[0031] In view of the fact that the distance L has a known value, it will be possible for the electronic processing device 43 to establish the magnitude of the support force F3 based on the formula [4] above and the first and second measuring values from the length measuring device 41 and the force sensor related to L2 and F2.

[0032] The invention is of course not in any way limited to the embodiments described above. On the contrary, several possibilities to modifications thereof should be apparent to a person skilled in the art without thereby deviating from the scope of the invention as defined in the appended claims.

Claims

1. A stabilizer leg arrangement for supporting a mobile working machine (1) against the ground, the stabilizer leg arrangement (2) comprising:

- a support beam (20) to be fixedly connected to a base structure (5) of the mobile working machine (1);
- an extension arm (21) mounted to the support beam (20), the extension arm (21) being telescopically extendible in order to allow an adjustment of the horizontal extension length thereof, wherein the extension arm (21) comprises an extension beam (22) which is telescopically mounted to the support beam (20) so as to be axially slidable in relation to the support beam (20) in the longitudinal direction of the extension arm (21) in order to vary the horizontal extension length thereof, and wherein this extension beam (22) extends into an interior space (23) of the support beam (20) through an entrance opening (24) provided at an end of the support beam;
- a stabilizer leg (26) mounted to the extension arm (21) at an outer end thereof, the stabilizer leg (26) being extendible in a vertical direction by means of an actuator (27), wherein a foot

plate (28) is arranged at a lower end of the stabilizer leg (26) so as to be vertically moveable, by a vertical extension of the stabilizer leg (26) under the effect of the actuator (27), from a raised position, in which the foot plate (28) is lifted from the ground, to a lowered operating position, in which the foot plate (28) is in supporting contact with the ground; and
 - a measuring system (40) for establishing the magnitude of the support force (F3) acting on said foot plate (28) in the operating position,

characterized in that the measuring system (40) comprises:

- a length measuring device (41) configured to generate a first measuring value that depends on the extension length of the extension arm (21);
- a force sensor (42) which is arranged on the support beam (20) in the vicinity of the entrance opening (24) and which is configured to generate a second measuring value that depends on the contact force (F2) between said extension beam (22) and the support beam (20) at the entrance opening (24); and
- an electronic processing device (43) configured to establish the magnitude of the support force (F3) based on said first and second measuring values.

2. A stabilizer leg arrangement according to claim 1, **characterized in that** the support beam (20) comprises a reinforcement collar (25) arranged around the entrance opening (24), wherein said force sensor (42) is arranged on the reinforcement collar (25).
3. A stabilizer leg arrangement according to claim 1 or 2, **characterized in that** said force sensor (42) comprises one or more strain gauges configured to sense the strain on a part of the support beam (20) in the vicinity of the entrance opening (24) caused by the contact force (F2) between said extension beam (22) and the support beam (20) at the entrance opening (24).
4. A stabilizer leg arrangement according to any of claims 1-3, **characterized in that** the actuator (27) is a hydraulic cylinder.
5. A stabilizer leg arrangement according to any of claims 1-4, **characterized in that** the electronic processing device (43) is configured to establish the magnitude of the support force (F3) based on the following formula:

$$F3 = \frac{F2 \cdot (L - L2)}{L}$$

where:

- F3 is the support force,
- F2 is the the contact force between said extension beam (22) and the support beam (20) at the entrance opening (24),
- L is the horizontal distance between a rear end (22a) of the said extension beam (22) and the centre axis of the stabilizer leg (26), and
- L2 is the horizontal distance between the entrance opening (24) and the centre axis of the stabilizer leg (26).

6. A mobile working machine comprising a vehicle (3) with a chassis, **characterized in that** the mobile working machine (1) comprises a stabilizer leg arrangement (2) according to any of claims 1-5, wherein the support beam (20) of the stabilizer leg arrangement (2) is fixedly connected to the chassis of the vehicle.
7. A mobile working machine according to claim 6, **characterized in that** the mobile working machine (1) comprises a hydraulic crane (6) mounted to the chassis of the vehicle.

Patentansprüche

1. Stabilisatorschenkelanordnung zum Stützen einer mobilen Arbeitsmaschine (1) gegenüber dem Boden, wobei die Stabilisatorschenkelanordnung (2) umfasst:
 - einen Stützträger (20) zur festen Verbindung mit einem Basisaufbau (5) der mobilen Arbeitsmaschine (1);
 - einen Ausfahrarm (21), der an dem Stützträger (20) montiert ist, wobei der Ausfahrarm (21) teleskopisch ausfahrbar ist, um eine Einstellung dessen horizontaler Ausfahrlänge zu ermöglichen, wobei der Ausfahrarm (21) einen Ausfahrträger (22) umfasst, der teleskopisch mit dem Stützträger (20) montiert ist, so dass er axial in Bezug auf den Stützträger (20) in der Längsrichtung des Ausfahrarmes (21) verschiebbar ist, um dessen horizontale Ausfahrlänge zu variieren, und wobei dieser Ausfahrträger (22) sich in einen Innenraum (23) des Stützträgers (20) durch eine Eintrittsöffnung (24) erstreckt, die an einem Ende des Stützträgers vorgesehen ist;
 - einen Stabilisatorschenkel (26), der an dem Ausfahrarm (21) an seinem äußeren Ende montiert ist, wobei der Stabilisatorschenkel (26) in

einer vertikalen Richtung mittels eines Aktors (27) ausfahrbar ist, wobei eine Fußplatte (28) an einem unteren Ende des Stabilisatorschenkels (26) zur vertikalen Bewegung durch ein vertikales Ausfahren des Stabilisatorschenkels (26) durch den Aktor (27) von einer erhöhten Position, in der die Fußplatte (28) von dem Boden angehoben ist, zu einer abgesenkten Betriebsposition angeordnet ist, in der die Fußplatte (28) in Stützkontakt mit dem Boden steht; und
 - ein Messsystem (40) zum Ermitteln der Größe der Stützkraft (F3), die auf die Fußplatte (28) in der Betriebsposition wirkt,

dadurch gekennzeichnet, dass das Messsystem (40) umfasst:

- eine Längenmessvorrichtung (41), die derart konfiguriert ist, einen ersten Messwert zu erzeugen, der von der Ausfahrlänge des Ausfahrarms (21) abhängt;
- einen Kraftsensor (42), der an dem Stützträger (20) in der Nähe der Eintrittsöffnung (24) angeordnet und derart konfiguriert ist, einen zweiten Messwert zu erzeugen, der von der Kontaktkraft (F2) zwischen dem Ausfahrträger (22) und dem Stützträger (20) an der Eintrittsöffnung (24) abhängt; und
- eine elektronische Verarbeitungsvorrichtung (43), die derart konfiguriert ist, die Größe der Stützkraft (F3) auf Grundlage des ersten und zweiten Messwerts herzustellen.

2. Stabilisatorschenkelanordnung nach Anspruch 1, **dadurch gekennzeichnet, dass** der Stützträger (20) einen Verstärkungsring (25) umfasst, der um die Eintrittsöffnung (24) angeordnet ist, wobei der Kraftsensor (42) an dem Verstärkungsring (25) angeordnet ist.

3. Stabilisatorschenkelanordnung nach einem der Ansprüche 1 oder 2, **dadurch gekennzeichnet, dass** der Kraftsensor (42) eine oder mehrere Dehnungsmesseinrichtungen umfasst, die derart konfiguriert sind, die Dehnung an einem Teil des Stützträgers (20) in der Nähe der Eintrittsöffnung (24) zu erfassen, die durch die Kontaktkraft (F2) zwischen dem Ausfahrträger (22) und dem Stützträger (20) an der Eintrittsöffnung (24) bewirkt wird.

4. Stabilisatorschenkelanordnung nach einem der Ansprüche 1 bis 3, **dadurch gekennzeichnet, dass** der Aktor (27) ein Hydraulikzylinder ist.

5. Stabilisatorschenkelanordnung nach einem der Ansprüche 1 bis 4, **dadurch gekennzeichnet, dass** die elektronische Verarbeitungsvorrichtung (43) derart konfiguriert ist, die Größe der Stützkraft (F3) auf

Grundlage der folgenden Formel herzustellen:

$$F3 = \frac{F2 \cdot (L - L2)}{L}$$

wobei:

- F3 die Stützkraft ist,
- F2 die Kontaktkraft zwischen dem Ausfahrträger (22) und dem Stützträger (20) an der Eintrittsöffnung (24) ist,
- L die horizontale Distanz zwischen einem rückwärtigen Ende (22a) des Ausfahrträgers (22) und der Zentralachse des Stabilisatorschenkels (26) ist, und
- L2 die horizontale Distanz zwischen der Eintrittsöffnung (24) und der Zentralachse des Stabilisatorschenkels (26) ist.

6. Mobile Arbeitsmaschine, die ein Fahrzeug (3) mit einem Fahrgestell umfasst, **dadurch gekennzeichnet, dass** die mobile Arbeitsmaschine (1) eine Stabilisatorschenkelanordnung (2) nach einem der Ansprüche 1 bis 5 umfasst, wobei der Stützträger (20) der Stabilisatorschenkelanordnung (2) fest mit dem Fahrgestell des Fahrzeugs verbunden ist.

7. Mobile Arbeitsmaschine nach Anspruch 6, **dadurch gekennzeichnet, dass** die mobile Arbeitsmaschine (1) einen Hydraulikkran (6) umfasst, der an dem Fahrgestell des Fahrzeugs befestigt ist.

Revendications

1. Agencement de pied stabilisateur pour supporter une machine de travail mobile (1) contre le sol, l'agencement de pied stabilisateur (2) comprenant :

- une poutre de support (20) à raccorder fixement à une structure de base (5) de la machine de travail mobile (1) ;
- un bras d'extension (21) monté sur la poutre de support (20), le bras d'extension (21) étant extensible de manière télescopique afin de permettre un ajustement de la longueur d'extension horizontale de celui-ci, dans lequel le bras d'extension (21) comprend une poutre d'extension (22) qui est montée de manière télescopique sur la poutre de support (20) de manière à pouvoir coulisser axialement par rapport à la poutre de support (20) dans le sens longitudinal du bras d'extension (21) afin de faire varier la longueur d'extension horizontale de celui-ci, et dans lequel cette poutre d'extension (22) s'étend dans un espace intérieur (23) de la poutre de support (20) à travers une ouverture d'entrée (24) pré-

vue à une extrémité de la poutre de support ;
 - un pied stabilisateur (26) monté sur le bras d'extension (21) à une extrémité extérieure de celui-ci, le pied stabilisateur (26) étant extensible dans un sens vertical au moyen d'un actionneur (27), dans lequel une embase (28) est agencée à une extrémité inférieure du pied stabilisateur (26) de manière à être mobile verticalement, par une extension verticale du pied stabilisateur (26) sous l'effet de l'actionneur (27), d'une position levée, à laquelle l'embase (28) est soulevée du sol, à une position de fonctionnement abaissée, à laquelle l'embase (28) est en contact de support avec le sol ; et
 - un système de mesure (40) pour établir la grandeur de la force de support (F3) agissant sur ladite embase (28) à la position de fonctionnement,

caractérisé en ce que le système de mesure (40) comprend :

- un dispositif de mesure de longueur (41) configuré pour générer une première valeur de mesure qui dépend de la longueur d'extension du bras d'extension (21) ;
 - un capteur de force (42) qui est agencé sur la poutre de support (20) à proximité de l'ouverture d'entrée (24) et qui est configuré pour générer une seconde valeur de mesure qui dépend de la force de contact (F2) entre ladite poutre d'extension (22) et la poutre de support (20) à l'ouverture d'entrée (24) ; et
 - un dispositif de traitement électronique (43) configuré pour établir la grandeur de la force de support (F3) sur la base desdites première et seconde valeurs de mesure.

2. Agencement de pied stabilisateur selon la revendication 1, **caractérisé en ce que** la poutre de support (20) comprend un collier de renfort (25) agencé autour de l'ouverture d'entrée (24), dans lequel ledit capteur de force (42) est agencé sur le collier de renfort (25).
3. Agencement de pied stabilisateur selon la revendication 1 ou 2, **caractérisé en ce que** ledit capteur de force (42) comprend une ou plusieurs jauges de contrainte configurées pour détecter la contrainte sur une partie de la poutre de support (20) à proximité de l'ouverture d'entrée (24) provoquée par la force de contact (F2) entre ladite poutre d'extension (22) et la poutre de support (20) à l'ouverture d'entrée (24).
4. Agencement de pied stabilisateur selon l'une quelconque des revendications 1 à 3, **caractérisé en ce que** l'actionneur (27) est un vérin hydraulique.

5. Agencement de pied stabilisateur selon l'une quelconque des revendications 1 à 4, **caractérisé en ce que** le dispositif de traitement électronique (43) est configuré pour établir la grandeur de la force de support (F3) sur la base de la formule suivante :

$$F3 = \frac{F2 \cdot (L - L2)}{L}$$

où :

- F3 est la force de support,
 - F2 est la force de contact entre ladite poutre d'extension (22) et la poutre de support (20) à l'ouverture d'entrée (24),
 - L est la distance horizontale entre une extrémité arrière (22a) de ladite poutre d'extension (22) et l'axe central du pied stabilisateur (26), et
 - L2 est la distance horizontale entre l'ouverture d'entrée (24) et l'axe central du pied stabilisateur (26) .

6. Machine de travail mobile comprenant un véhicule (3) avec un châssis, **caractérisée en ce que** la machine de travail mobile (1) comprend un agencement de pied stabilisateur (2) selon l'une quelconque des revendications 1 à 5, dans laquelle la poutre de support (20) de l'agencement de pied stabilisateur (2) est raccordée fixement au châssis du véhicule.
7. Machine de travail mobile selon la revendication 6, **caractérisée en ce que** la machine de travail mobile (1) comprend une grue hydraulique (6) montée sur le châssis du véhicule.

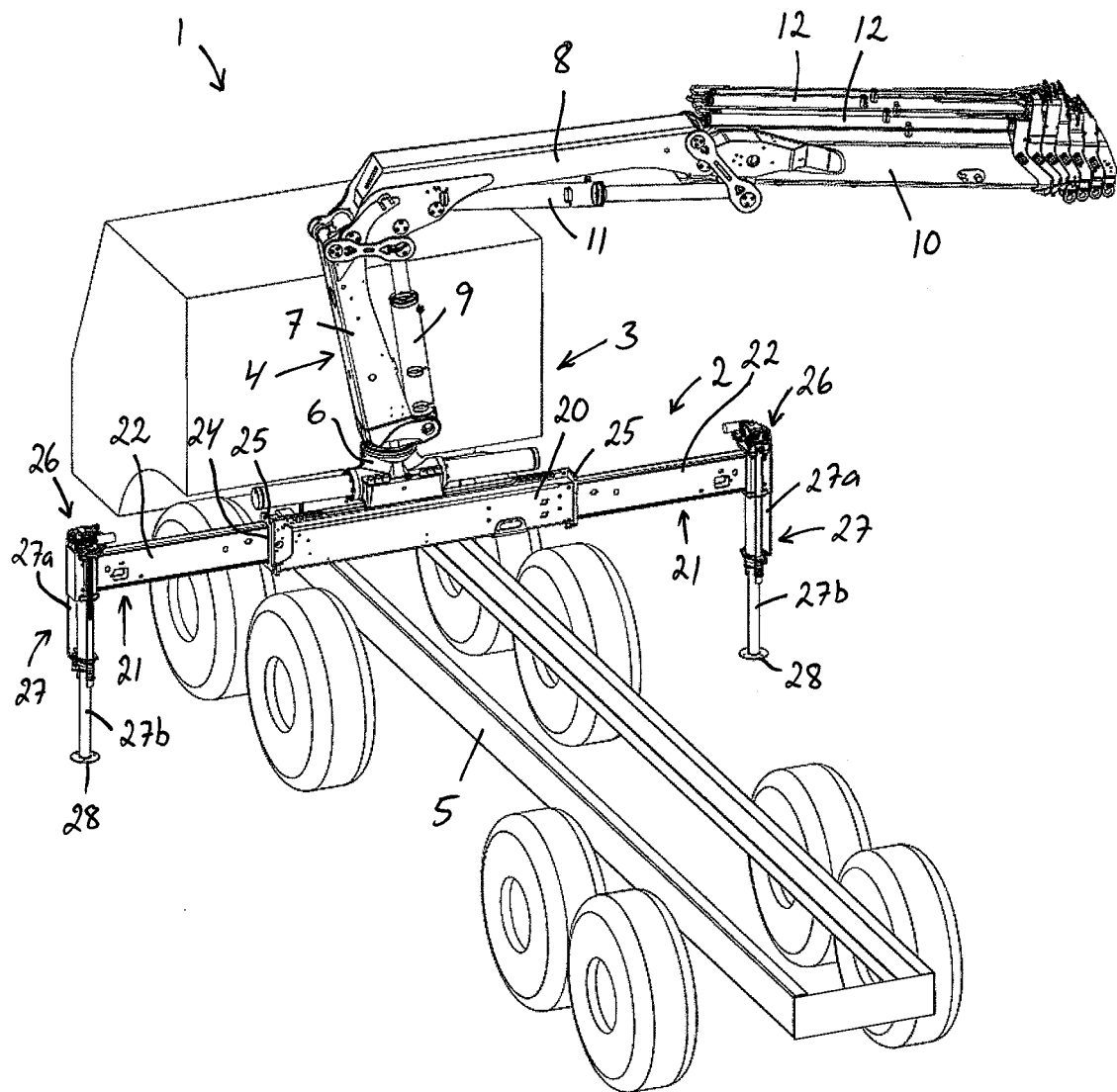
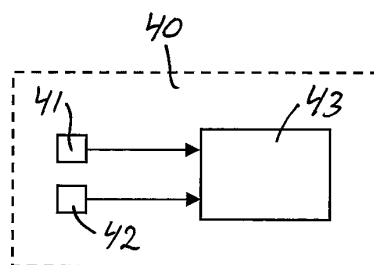
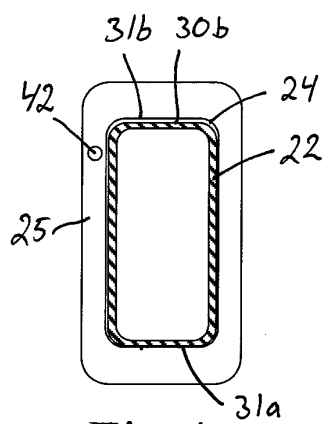
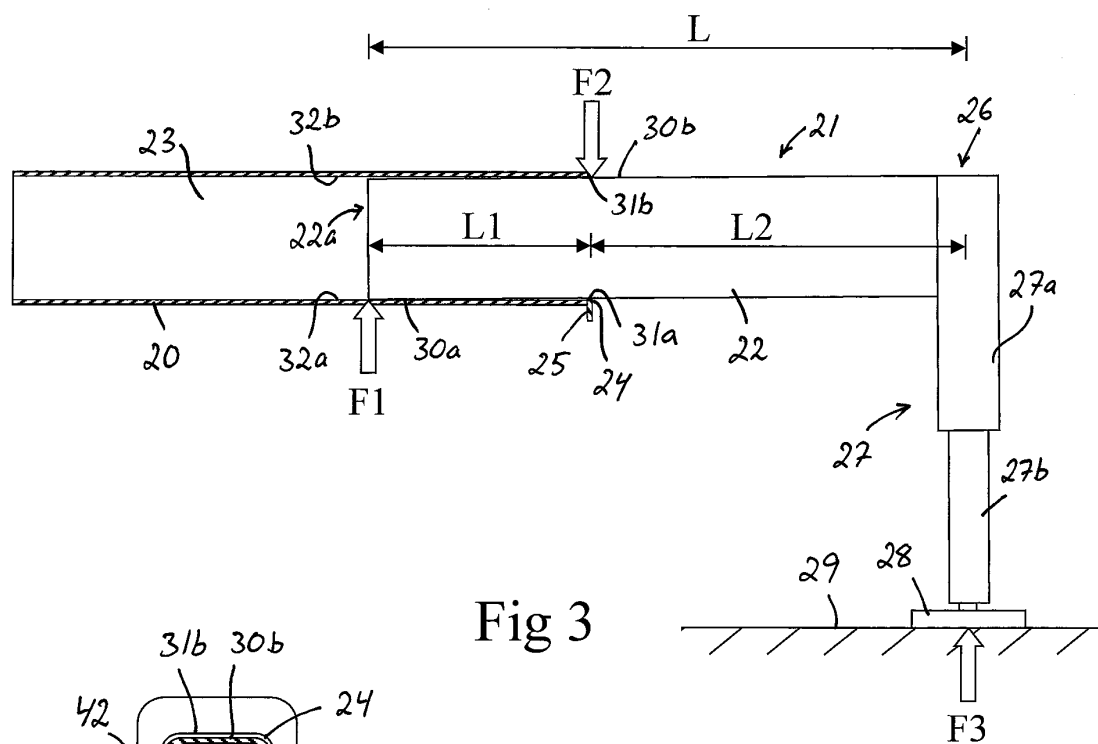
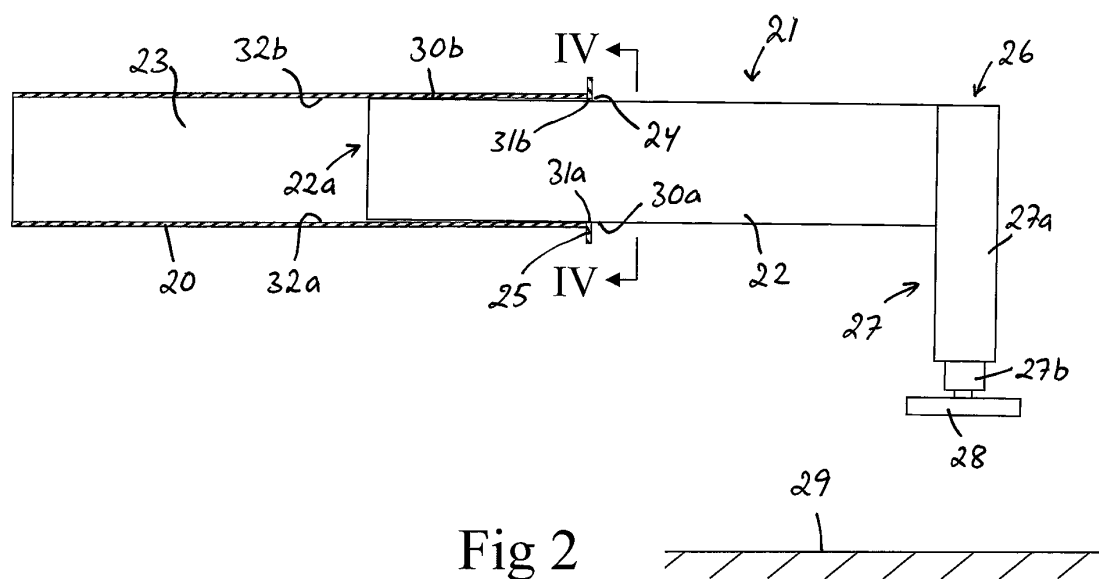


Fig 1



REFERENCES CITED IN THE DESCRIPTION

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