

Filed Sept. 17, 1953

CHECK IMPRINTING PRESS

3 Sheets-Sheet 1



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Nov. 12, 1957

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2,812,940

CHECK IMPRINTING PRESS

Filed Sept. 17, 1953

3 Sheets-Sheet 2

FIG. 3.

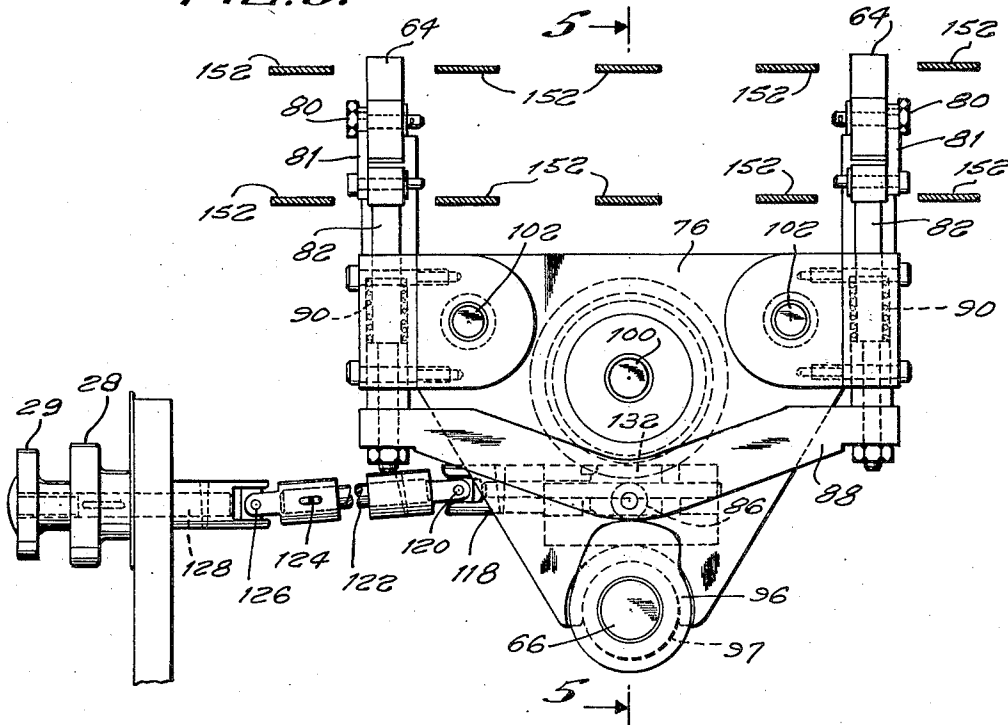
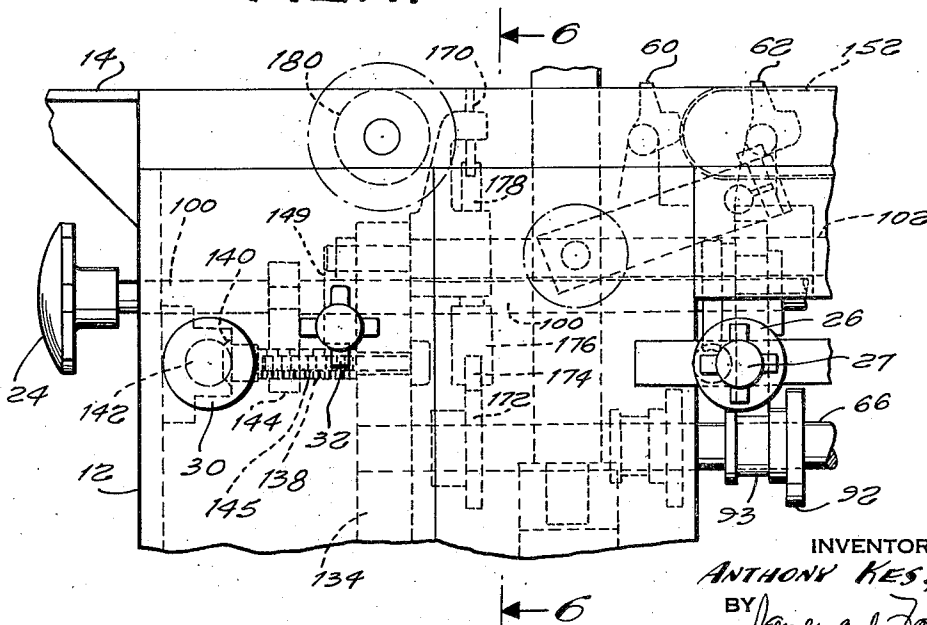


FIG. 4.



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Filed Sept. 17, 1953

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FIG. 7.

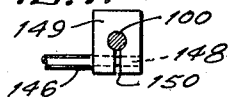
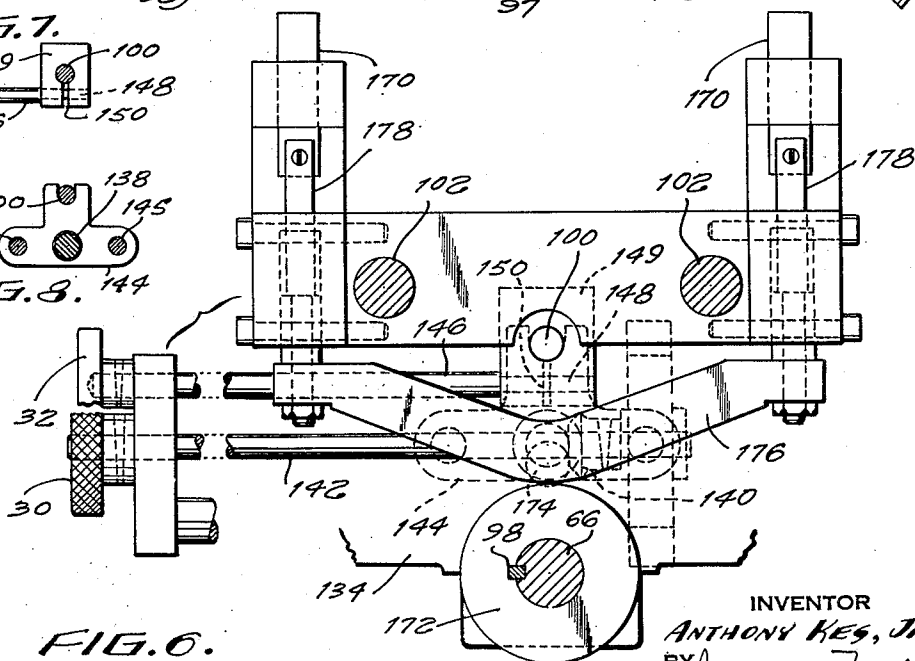
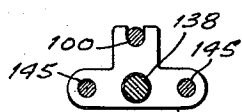


FIG. 8. 149



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CHECK IMPRINTING PRESS

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Application September 17, 1953, Serial No. 380,797

20 Claims. (Cl. 271—50)

This invention relates to check imprinting presses, and more particularly to the paper stop mechanism for such presses.

Most checkbooks have multiple checks on a sheet, typically three collateral checks, or five collateral checks, with suitable perforations for tearing the checks from the stubs, and from one another at their adjacent side edges. Ordinarily sheets are preliminarily printed and perforated all alike, without attempting to serially number the checks and stubs, nor to imprint the name of the depositor. At a later time, preparatory to assembling sheets into a checkbook for a specific depositor, the added information is supplied by a second printing operation. In some cases where the bank has branches, the name of the branch may be added in the second printing operation. Presses for the second printing operation are called "check imprinting presses," and the printing chase includes a series of digit wheels which are advanced automatically one digit for each printing operation. For this purpose the checks are printed one at a time, so that if a single sheet has three checks, the sheet is advanced step by step and printed three times. This may be done by using a series of paper stops disposed one after the other in the direction of paper feed, together with suitable mechanism to make the stops sequentially effective.

One difficulty which arises is that the check size is not standardized, and the height of the checks to be printed may range from, say $2\frac{3}{4}$ " to $3\frac{1}{2}$ ". Thus the spacing between the stops must be correspondingly varied, and if the spacing between the first and second stop is increased say $\frac{1}{4}$ ", the spacing between the second and third stops must be increased $\frac{1}{4}$ ", representing a change of $\frac{1}{2}$ " for the third stop relative to the first.

The primary object of the present invention is to generally improve check imprinting presses, and more particularly to overcome the foregoing difficulty by simplifying the adjustment of the spacing between stops. A more specific object is to provide a single control element which will change all of the spacings by a uniform amount, the position of a third stop being changed double the change for a second stop, and so on.

Because of possible variations in the location of the printing of one check relative to another when performing the first printing operation, it is desirable to provide a minor adjustment or compensation in which each stop may be adjusted independently of the others, and to make this possible constitutes a further object of the present invention.

Moreover, the printing of the first check may vary relative to the top edge of the sheet of checks, this action being a factor resulting from either a change in height of the check, or a change in the printing layout of the check itself. This difference carries through to the other checks of the sheet. In accordance with a further feature and object of the present invention I provide for simultaneous adjustment of all of the stops and associated mechanism, relative to the printing chase, thus providing for overall control of registration.

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To accomplish the foregoing general objects, and other more specific objects which will hereinafter appear, my invention resides in the check imprinting press elements, and their relation one to another, as are hereinafter more particularly described in the following specification. The specification is accompanied by drawings, in which:

Fig. 1 is a perspective view showing a check imprinting press embodying features of my invention;

Fig. 2 is a side elevation of the same with a side plate removed to expose some of the mechanism of the press;

Fig. 3 is a detailed section drawn to enlarged scale and drawn approximately in the plane of the line 3—3 of Fig. 2;

Fig. 4 is a fragmentary side elevation corresponding to the upper lefthand part of Fig. 2 but drawn to larger scale;

Fig. 5 is a section taken approximately in the plane of the line 5—5 of Fig. 3;

Fig. 6 is a section taken approximately in the plane of the line 6—6 of Fig. 4;

Fig. 7 is a detail of a lock for one of the adjustments; and

Fig. 8 is a detail explanatory of a part shown in Figs. 4 and 6.

Referring to the drawing, and more particularly to Fig. 1, the check imprinting press comprises a housing 12, and a table 14 from which the sheets of checks are fed either manually, or automatically by means of appropriate mechanism such as a suction feed wheel. The checks are printed by a suitable printing chase housed at 16, it being understood that this is reciprocated vertically down to the paper for the printing operation. The inking mechanism is housed at 18, and may be conventional. Treadle 20 serves to stop or start operation of the press, while hand wheel 22 may be used to turn over the printing shaft and other mechanism of the press when adjusting the same.

In accordance with the present invention I provide a main adjusting knob 24, rotation of which serves to vary the spacing between stops, and this variation is cumulative, so that a third stop is moved twice as far as a second stop, and so on. The particular machine here illustrated is intended to print sheets having three checks and stubs, but the principles of the invention may be applied to sheets having a greater number of checks and stubs, for example five.

An adjusting knob 26 serves to provide a minor or vernier adjustment of the second stop, in either direction, independently of the first and third stops. A similar adjusting knob 28 affords adjustment of the third stop independently of the first and second stops. The adjustments may be locked by suitable means, in this case concentric locking knobs 27 and 29 respectively. These adjustments do not interfere with the main cumulative adjustment produced by the adjusting knob 24.

Another adjusting knob 30 at the side of the machine provides for overall registration of the sheet relative to the printing chase. In this case rotation of the knob produces a common or bodily motion of all of the stops and their associated adjusting means. The adjustment may be locked by a special locking knob 32. Here again the adjustment is independent of the main cumulative spacing adjustment produced by knob 24, and also of the independent relative adjustments of the stops produced by knobs 26 and 28.

Referring now to Fig. 2, the main printing shaft is shown at 34. This is suitably connected to produce vertical reciprocation of the chase 16 through appropriate vertical slide bars 36, and there is one printing operation for each rotation of shaft 34. The press is driven by an electric motor 38, which drives a counter-shaft 40 through a V belt drive 42 and a spring loaded variable pitch pulley

44. The speed of operation of the press may be changed by shifting the motor bodily by means of a screw 46 and crank 48, and if the motor is moved to the right the sides of the pulley 44 are sprung apart, so that the pulley assumes a new effective diameter which is smaller, as indicated at 44', thus reducing the operating speed of the press.

The printing shaft 34 reciprocates a rod 50 which at its upper end is a gear rack which in turn oscillates a pinion 52. This oscillates a gear 54 which in turn meshes with a rack 56 carrying an inking roller 58. This in turn is in contact with a train of inking rollers leading to an ink fountain, all in the ink distribution housing 18. It will be understood that in the course of each printing operation, during a dwell period when the chase 16 is raised, the inking roller 58 is moved forward and back again beneath the chase in order to ink the same.

The paper feed is arrested by a series of paper stops indicated schematically at 60, 62, 64. These are spaced apart by an amount corresponding to the height of an individual check, and for different banks this height may vary from, say, $2\frac{3}{4}$ " to $3\frac{1}{2}$ ". The stops are retracted sequentially by cams mounted on a cam shaft 66, which is driven through mitre gears 68, by a shaft 70, geared to the printing shaft 34 by a sprocket chain 71 and sprocket wheels for a positive speed relation of 1:3, that is, cam shaft 66 turns one revolution for three revolutions of printing shaft 34. If there were five stops there would be five cams, and the gear reduction would be 5:1 instead of 3:1.

Referring now to Fig. 5, the stops 60, 62 and 64 are generally alike, being pivoted on and carried by supports 72, 74, and 76. Stop 64 is pivoted at 78, and is connected at 80 to a generally upright link 81 and thence to a slide 82, which at its lower end carries a bearing 84 for a cam follower or roller 86 which rests on cam 96. The roller is normally urged downward by means of a compression spring 90. It will be evident that the stop is normally retracted or lowered as shown by the broken line position 60' for the stop 60, and that it is raised to effective or stopping position as shown at 64 only when forced upward by the cam 96.

Similar mechanism is provided for each of the stops, and each has its own cam as indicated at 92, 94 and 96. The cams are all carried on the cam shaft 66 previously referred to, and it will be noted that they are splined or slidably keyed to the shaft as indicated at 98, so that the cams may slide along or shift axially of the shaft.

Referring now to Fig. 3, it will be seen that there are a pair of stops 64 with vertical operating rods 82 and compression springs 90. The lower ends of the rods 82 are connected to a common yoke 88, and the cam roller 86 is located centrally of the yoke 88. The cam shaft 66 and cam 96 are directly beneath the roller 86, and are disposed symmetrically with respect to the stops 64. The main support 76 is in the form of an inverted triangle, and it receives therethrough a member 100, which is the main adjusting means for cumulative adjustment of the stop spacing. The support 76 itself is slidably carried on a pair of spaced parallel horizontal slide rods 102.

Reverting now to Fig. 5, the main adjusting member 100 is a pair of screws preferably arranged coaxially or in tandem. Specifically, the shaft 100 is threaded at 104 to provide a first screw which threadedly engages the support 74, and is threaded at 106 to provide a second screw which threadedly engages the support 76. The threads 104 and 106 differ, and the lead of the screw 106 is made twice the lead of the screw 104. The support 72 of the first stop is preferably immovably related (in axial direction) to the shaft 100, as is indicated by the step or shoulder 108 and collar 110. It will thus be seen that rotation of shaft 100 moves the support 76 and consequently the stops 64 twice as far, relative to the stop 60, as it moves the support 74 with its stops 62. This is

indicated by the fragmentary broken-line showing of the stops at 60'', 62'', and 64''.

In Fig. 5 it will be observed that each of the supports 72, 74 and 76 is shaped with a yoke-like half bearing at its lower end which is received in a corresponding journal formed in the hub of the cams. Thus cam 92 has a journal 93; cam 94 has a journal 95; and cam 96 has a journal 97. This inter-relation of the parts is also clearly shown in Fig. 3 in which it will be seen how the lower end of support 76 is received in journal 97 of cam 96. It will be evident that with this arrangement the cams are shifted along the cam shaft 66 as the supports are shifted by the lead screw shaft 100. The cam rollers, of course, move with the supports because the yokes 88 are carried by the supports, all as previously described.

Referring to Fig. 4, it will be seen that the forward end of shaft 100 projects through the front of the casing 12 of the press and there receives the main adjusting knob 24 previously referred to.

Reverting now to Fig. 5, the support 74 does not directly engage the thread 104. Instead the support 74 rotatably carries a nut 112 which is immovable axially relative to the support. This nut in turn carries and may be rotated by a worm gear 114 which meshes with a worm 116. This may be used for independent minor adjustment. There is similar mechanism for the support 76, which similarly has a rotatable nut and worm gear 130, meshing with a worm 132.

In Fig. 3 it will be seen that the worm 132 is connected to a control knob 28 located at the side of the press, this connection preferably being by means of a flexible shaft, which, in the present case, comprises a shaft 118, a universal joint 120, an intermediate shaft 122, a slidable connection 124, a second universal joint 126, and a stationary shaft 128. This carries and may be turned by the knob 28. It will be evident that rotation of knob 28 drives the worm 116, which in turn slightly turns the nut 112 (Fig. 5), thus slightly shifting the position of stop 64 relative to the stops 60 and 62. This adjustment is independent of the other stops, and the flexible shaft arrangement accommodates any main movement or adjustment of the stops produced by rotation of the knobs 24 and 30 previously referred to in connection with Fig. 1.

The adjusting knob 28 is accompanied by a concentric knob 29 which is arranged to act as a locking knob to lock the adjustment of the flexible shaft. This lock may be any of a number of conventional locks used for the purpose, and may, for example, be threaded to wedge the hub of joint 126 against the frame of the machine. Moreover, it will be understood that a flexible shaft or universally jointed shaft similar to that shown in Fig. 3 is provided for the worm 116 with the associated knobs 26 and 27.

Referring now to Fig. 4 of the drawing, it will be seen that the slide rods 102 terminate at their forward ends in a stationary plate 134. The righthand or rear ends of the rods 102 are supported in a somewhat similar stationary plate 136 shown in Fig. 5 of the drawing. To provide for overall bodily registration of the stops relative to the printing chase I provide a screw 138 (Fig. 4) the rear end of which is rotatably carried in the plate 134. At its forward end this screw is rotated by mitre gears 140 turned by a shaft 142, which at its outer end carries the adjusting knob 30 previously referred to. Screw 138 carries a nut 144, the upper end of which is connected to a reduced or "necked" part of the shaft 100, in order to cause axial movement thereof. The nut 144 has the shape of an inverted T, as best shown in Fig. 8, and also in Fig. 6. It is guided on a pair of slide rods 145.

It will be evident that the resulting movement of shaft 100 causes a simultaneous equal movement of all three of the stops, as will be clear from inspection of Fig. 5, it being understood that the shaft 100 is axially slidable in the stationary plates 134 (Fig. 4) and 136 (Fig. 5).

The resulting adjustment may be locked by rotating a

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lock knob 32, and referring now to Fig. 6, it will be seen that knob 32 turns a shaft 146 the end of which is threaded at 148 and acts as a clamping screw for a brake 149 which is split at 150. These parts are shown separately in Fig. 7. The brake 149 is secured to stationary plate 134 (Fig. 4), and thus locks the adjusting shaft 100 against rotation and against axial movement. Thus the lock knob 32 locks both knob 24 and knob 30.

The feed of the sheet through the press is by means of slip belts or so-called "transport tapes" disposed beneath the sheet, and weighted balls resting thereabove on top of the sheet. One of the tapes is schematically indicated at 152 in Fig. 5. In the present case there are five such tapes, as is most clearly shown in section in Fig. 3. The stops when raised come higher than the tapes, as shown in Figs. 3, 4, and 5. To hold the sheets downward for frictional contact with the tapes I employ steel balls which are freely rotatable in holders disposed directly above the tapes. These are not shown in the drawing.

There are preliminary gate stops 170 (Figs. 4 and 6) which are moved vertically by a cam 172, bearing against a roller 174 carried at the center of a cross yoke 176 which at its ends is connected to vertical slide rods 178, which in turn carry the gate stops 170. Whether the sheet is fed manually, or automatically by a suction wheel, it preliminarily comes up against the gate stops 170. At this time the sheet rests on a rubber roller 180 (Figs. 2 and 4), and is held downward thereon by weighty freely rotatable steel balls, not shown. The gate stops are pulled down about the same time that the last of the locking stops 64 are pulled back, and thus the sheet is automatically fed up to the first stops 60, which are raised at that time. The shape of cam 172 is not shown.

The three cams 92, 94 and 96 permit the stops to rock back or retract at intervals of 120°. However, the stops need not be raised at intervals of 120°, and in practice it is found preferable to take advantage of the fact that the second stop may be raised for part or all of the time while the first stop is raised, and that the third stop may be raised for part of the time while the first and second stops are raised. Thus the stops may be safely elevated and already waiting in position for the sheet when the sheet is advanced by the transport belts.

The rubber roller 180 is used ahead of the printing chase 16 (Fig. 2), and the transport tapes 152 begin behind the printing chase 16, in order that the printing plate may bear downward against a sheet resting on a solid table surface, rather than on yieldable tapes.

There is a discharge table 160 (Fig. 2) on which the printed sheets are deposited. The press includes a "fly leaf" which turns the sheets over so that they are inverted on table 160 in order to keep the serial numbering in increasing sequence. The fly leaf and the mechanism for operating the same are not shown, but, of course, are geared for operation in appropriate time relation to the delivery of the printed sheets. There are also adjustable side guides for the sheets, not shown. It will be understood that there is appropriate linkage, not shown, whereby the digit wheels in the printing chase 16 are advanced one digit for each reciprocation of the printing chase.

It is believed that the construction and operation of my improved check imprinting press, as well as the advantages thereof, will be apparent from the foregoing detailed description. My improved mechanism provides micro-metrically accurate registration by means of fingertip controls which are readily accessible outside the casing of the machine, so that there is no need to open the casing when changing from one kind of check to a completely different kind of check, or when correcting for changes in the printing on the sheets received from the first printer.

It will be understood that while I have shown the invention applied to a machine for imprinting sheets having three checks each, the same constructional features may be applied to machines for imprinting sheets having a greater number, typically five checks each. In

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such case there will be five stops and four screws, with the lead of the second screw double that of the first; the lead of the third screw triple that of the first; and the lead of the fourth screw quadruple that of the first. All of the supports, except the first, preferably have adjustable nuts using, for example, the worm gears and worms with flexible shafts, as here shown. All five of the stops are bodily shifted with the main adjusting screw when making the registration adjustment by means of the knob 30.

It will also be apparent that while I have shown and described the invention in a preferred form, changes may be made in the structure shown without departing from the scope of the invention, as sought to be defined in the following claims. In the claims the reference to a cam roller is not intended to exclude other forms of cam follower. The movement of the second and third stops is in some cases referred to as being produced by first and second screws, it being understood that separate parallel screws may be employed instead of tandem screws, and that in such case they need not differ in lead, and instead could differ in rotative speed, as by being geared together in 2 to 1 ratio. In the claims the reference to flexible shafts leading to control knobs is intended to apply to either truly flexible shafts, or sectional shafts with universal joints as shown.

I claim:

1. In a multiple check imprinting press having a printing station and means to feed a sheet of paper comprising three or more collateral checks, said means feeding said paper across said printing station from check to check, a mechanism for adjusting for a change in check size, said mechanism comprising a series of paper stops disposed in the direction of paper feed, means to vary the spacing between a first and second stop, means to vary the spacing between the first and a third stop, and a single control means to simultaneously vary the latter spacing twice as much as the former spacing.

2. In a multiple check imprinting press having a printing station and means to feed a sheet of paper comprising three or more collateral checks, said means feeding said paper across said printing station from check to check, a mechanism for adjusting for a change in check size, said mechanism comprising a plurality of retractable paper stops disposed and operated in sequence in the direction of paper feed, a support carrying the second stop and movable toward or away from the first stop, a support carrying the third stop and movable toward and away from the first stop, a screw extending in the direction of paper feed and engaging the second stop carrying support, a screw extending in the direction of paper feed and engaging the third stop carrying support, a single control handle to rotate both screws, and means whereby said second screw changes the spacing between the first and third stops an amount double the amount of the change produced by the first screw between the first and second stops.

3. In a multiple check imprinting press having a printing station and means to feed a sheet of paper comprising three or more collateral checks, said means feeding said paper across said printing station from check to check, a mechanism for adjusting for a change in check size, said mechanism comprising a plurality of retractable paper stops disposed and operated in sequence in the direction of paper feed, a support carrying the second stop and movable toward or away from the first stop, a support carrying the third stop and movable toward and away from the first stop, a screw extending in the direction of paper feed, a thread on said screw engaging the second stop carrying support, and a thread on said screw engaging the third stop carrying support, said latter thread having a lead twice the lead of the first thread, whereby rotation of said screw changes the spacing between the first and second stops and simultaneously changes the spacing between the first and third stops an amount

double the amount of the change between the first and second stops.

4. In a multiple check imprinting press having a printing station and means to feed a sheet of paper comprising three or more collateral checks, said means feeding said paper across said printing station from check to check, a single control mechanism for adjusting for a change in check size, said mechanism comprising a series of retractable stops disposed in the direction of paper feed, a cam shaft disposed in the direction of paper feed, a first cam and roller to rise or retract a first stop, a second cam and roller to raise or retract the second stop, a third cam and roller to raise or retract the third stop, said second and third cams being slidable axially of said cam shaft, a support carrying said second stop for movement toward or away from the first stop, the said support having means to move with it the second cam and roller, a support carrying said third stop for movement toward and away from the first stop, the said support having means to move with it the third cam and roller, and a single control means to move the second support in either direction and to simultaneously move the third support twice as far as the second support.

5. In a multiple check imprinting press having a printing station and means to feed a sheet of paper comprising three or more collateral checks, said means feeding said paper across said printing station from check to check, a single control mechanism for adjusting for a change in check size, said mechanism comprising a series of retractable stops disposed in the direction of paper feed, a cam shaft disposed in the direction of paper feed, a first cam and roller to raise or retract a first stop, a second cam and roller to raise or retract the second stop, a third cam and roller to raise or retract the third stop, said second and third cams being slidable axially of said cam shaft, a support carrying said second stop for movement toward or away from the first stop, the said support having means to move with it the second cam and roller, a support carrying said third stop for movement toward and away from the first stop, the said support having means to move with it the third cam and roller, a screw extending parallel to the cam shaft and threadedly engaging said second support, a screw extending parallel to the cam shaft and threadedly engaging the third support, and means whereby the second screw moves the third support simultaneously with the first screw and double the amount the first screw moves the second support.

6. In a multiple check imprinting press having a printing station and means to feed a sheet of paper comprising three or more collateral checks, said means feeding said paper across said printing station from check to check, a single control mechanism for adjusting for a change in check size, said mechanism comprising a series of retractable stops disposed in the direction of paper feed, a cam shaft disposed in the direction of paper feed, a first cam and roller to raise or retract a first stop, a second cam and roller to raise or retract the second stop, a third cam and roller to raise or retract the third stop, said second and third cams being slidable axially of said cam shaft, a support carrying said second stop for movement toward or away from the first stop, the said support having means to move with it the second cam and roller, a support carrying said third stop for movement toward and away from the first stop, the said support having means to move with it the third cam and roller, and a screw extending parallel to the cam shaft and threadedly engaging said second and third supports, the thread at said third support having a lead double the lead of the thread at the second support.

7. A check imprinting press as defined in claim 2, in which the second and third supports each include a rotatable nut engaging the screw, gear teeth on said nut, and a control knob and gear means meshing with each of said nuts, whereby relative adjustment of each of the

said second and third stops may be made independently of the others.

8. A check imprinting press as defined in claim 5, in which the second and third supports each include a rotatable nut engaging the screw, gear teeth on said nut, and a control knob and gear means meshing with each of said nuts, whereby relative adjustment of each of the said second and third stops may be made independently of the others.

9. A check imprinting press as defined in claim 3, in which the second and third supports each includes a rotatable nut engaging the screw, a worm gear on said nut, a worm engaging said worm gear, a flexible shaft, and a control knob for relative adjustment of each of said stops independently of the others.

10. A check imprinting press as defined in claim 6, in which the second and third supports each includes a rotatable nut engaging the screw, a worm gear on said nut, a worm engaging said worm gear, a flexible shaft, and a control knob for relative adjustment of each of said stops independently of the others.

11. A check imprinting press as defined in claim 1, in which all of the stops and associated adjusting means are bodily movable in unison for a short distance in the direction of paper feed, support means affording such bodily movement, and adjusting means for moving the same in the direction of paper feed for overall registration control.

12. A check imprinting press as defined in claim 4, in which all of the stops and associated adjusting means are bodily movable in unison for a short distance in the direction of paper feed, support means affording such bodily movement, and adjusting means for moving the same in the direction of paper feed for overall registration control.

13. A check imprinting press as defined in claim 2, in which all of the stops, supports, and check size adjusting screws are bodily movable in unison for a short distance in the direction of paper feed, support means affording such bodily movement, and adjusting means for moving the same in the direction of paper feed for overall registration control.

14. A check imprinting press as defined in claim 5, in which all of the stops, supports, and check size adjusting screws are bodily movable in unison for a short distance in the direction of paper feed, support means affording such bodily movement, and adjusting means for moving the same in the direction of paper feed for overall registration control.

15. A check imprinting press as defined in claim 3, in which all of the stops and supports are movable with the check size adjusting screws for a short distance in the direction of paper feed, support means affording such bodily movement, a control knob and screw mechanism for moving said screws in the direction of paper feed for overall registration control, and means to help lock the adjustment provided.

16. A check imprinting press as defined in claim 6, in which all of the stops and supports are movable with the check size adjusting screws for a short distance in the direction of paper feed, support means affording such bodily movement, a control knob and screw mechanism for moving said screws in the direction of paper feed for overall registration control, and means to help lock the adjustment provided.

17. A check imprinting press as defined in claim 2, in which the second and third supports each include a rotatable nut engaging the screw, gear teeth on said nut, and a control knob and gearing meshing with said teeth whereby relative independent adjustment of each of the second and third stops is available, and in which all of the stops, supports, and check size adjusting screws are bodily movable in unison for a short distance in the direction of paper feed, support means affording such bodily movement, and adjustment means for moving the same

in the direction of paper feed for overall registration control.

18. A check imprinting press as defined in claim 3, in which the second and third supports each include a rotatable nut engaging the screw, gear teeth on said nut, and a control knob and gearing meshing with said teeth whereby relative independent adjustment of each of the second and third stops is available, and in which all of the stops, supports, and check size adjusting screws are bodily movable in unison for a short distance in the direction of paper feed, support means affording such bodily movement, and adjustment means for moving the same in the direction of paper feed for overall registration control.

19. A check imprinting press as defined in claim 5, in which the second and third supports each include a rotatable nut engaging the screw, a worm gear on said nut, a worm engaging said worm gear, and a flexible shaft between said worm and a control knob at the side of the press, whereby relative independent adjustment of each of the second and third stops is available, and in which the stops, supports and cams are movable with the check size adjusting screws for a short distance in the direction of paper feed, support means affording such bodily movement, a control knob and screw mechanism

for moving said screws in the direction of paper feed for overall registration control, and means to help lock the adjustment provided.

20. A check imprinting press as defined in claim 6, in which the second and third supports each include a rotatable nut engaging the screw, a worm gear on said nut, a worm engaging said worm gear, and a flexible shaft between said worm and a control knob at the side of the press, whereby relative independent adjustment of each of the second and third stops is available, and in which the stops, supports and cams are movable with the check size adjusting screws for a short distance in the direction of paper feed, support means affording such bodily movement, a control knob and screw mechanism for moving said screws in the direction of paper feed for overall registration control, and means to help lock the adjustment provided.

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