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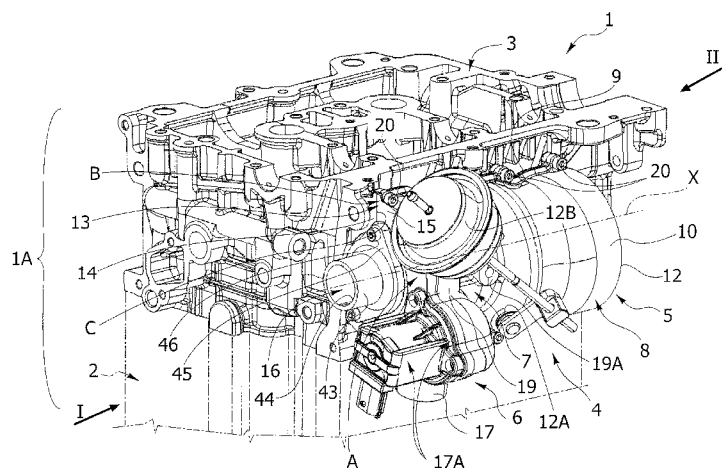
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(54) Title: Supercharged internal combustion engine

FIG. 1



(57) Abstract: A supercharged internal combustion engine (1, 100, 200) comprising: - a motor unit (1A, 101, 201) including a head (3, 103, 203) and an exhaust manifold (36, 236), a turbocharger assembly (4, 104, 204) fluid dynamically connected to said exhaust manifold (36, 236), wherein said turbocharger assembly (4, 104, 204) includes a turbine (5, 105, 205), a central body (7, 107, 207) and a compressor (6, 106, 206). The turbocharger assembly (4, 104, 204) comprises a lubrication channel (24A, 224A) for the passage of a lubricating fluid hydraulically connected to a lubrication circuit of said motor unit (1A, 101, 201) of said internal combustion engine (1, 100, 200). The turbine (5, 105, 205) comprises a jacket (34, 234), provided at least in part in a body (8, 108, 208) thereof, arranged for the passage of a cooling fluid and in hydraulic communication with an inlet channel (41, 234A) and an outlet channel (42, 234B) hydraulically connected to a cooling circuit (39, 40; 239, 240) of said motor unit (1A, 101, 201) of said internal combustion engine (1, 100, 200). Said inlet channel (41, 234A), outlet channel (42, 234B) and lubrication channel (24A, 224A) being integrated in said turbocharger assembly (4, 104, 204) in correspondence of a connection interface between said turbocharger assembly (4, 104, 204) and the motor unit (1A, 101, 201).



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"Supercharged internal combustion engine"*******Field of the invention

The present invention refers to a supercharged
5 internal combustion engine comprising:

- a motor unit including a head and an exhaust manifold,
- a turbocharger assembly fluid dynamically connected to said exhaust manifold, wherein said
10 turbocharger assembly includes a turbine, a central body and a compressor.

Such a supercharged internal combustion engine is known, for example, from US 2009/151327 A1, US-A-4,716,735, JP-A-2006-063851, JP-A-2006-249945, JP-A-
15 2006-194227, JP-A-2002-303145, DE 31 33 953 A1, JP-A-57-052624 and US-A-3,948,052.

Description of the prior art and general technical problem

Supercharging internal combustion engines by means
20 of turbocharger today plays a key role in the automotive industry due to the demand, by the market, for vehicles with lower environmental impact, low fuel consumption and at the same time offering good performance. In particular, currently the most common
25 design trend among automotive producers is that of "downsizing", i.e. the reduction of the engine displacement, hence requiring, in order to guarantee good performance for vehicles on which the engines are installed, supercharging the same engines.

30 However, supercharging by means of turbocharger entails several well known drawbacks. Apart from, for example, the well known problems related to knock control typical of spark ignition engines, the problems regarding the hydraulic connection between the
35 turbocharger and the internal combustion engine for the

circulation of oil and possibly cooling water within the central body of the turbocharger itself are of considerable importance.

In fact, the internal combustion engines of the known type generally comprise one or more pipings having an end connected hydraulically and mechanically to the engine itself, for example to a cooling circuit or to a lubrication circuit, and a second end connected hydraulically and mechanically to inlet/outlet ports located on the central body of the turbocharger. The cost of such connections is quite high because each connection between an internal combustion engine and turbocharger implies the use of fittings, gaskets and fastening elements.

Nevertheless, on spark ignition engines, the turbine body is made of high resistance steel with high contents of nickel (for example Nimonic® steel). Such steel is very expensive and the higher the nickel content the more expensive the steel. The higher the temperature of the exhaust gases flowing into the turbine, the higher the nickel amount required in such steels.

Furthermore, there arises a related drawback typical of turbocharged engines: the body of the turbine, not cooled, reaches extremely high working temperatures, hence, combined with the considerable overall dimensions of the turbocharger assembly, it makes the installation of the latter complex, because it is necessary to envisage the use of protection panels for protecting the surrounding components from contact with the body of the turbine or from overexposure to a heat flow coming therefrom.

Lastly, if on one hand there are drawbacks related to the high cost of some materials, to the working temperatures and upon installation of the turbocharger

assembly, on the other hand there are problems regarding the high number of components required for the coupling of a turbocharger assembly to an internal combustion engine, as well as the high number of components that form the turbocharger assembly itself.

Object of the invention

The object of the present invention is that of overcoming the previously described drawbacks. In particular, the object of the invention is that of reducing the cost and the number of components required for producing and for coupling a turbocharger to the motor unit of an internal combustion engine, as well as that of optimising the operation of the turbo assembly in transient conditions of the engine following a cold start.

Summary of the invention

The object of the present invention is attained by a turbocharged internal combustion engine having the features forming the subject of the claims that follow, which form an integral part of the technical disclosure provided herein in relation to the invention.

In particular, the object of the present invention is attained by an internal combustion engine having all the characteristics listed at the beginning of the present description and further characterised in that:

- said turbocharger assembly comprises a lubrication channel for the passage of a lubricating fluid hydraulically connected to a lubrication circuit of said motor unit of said internal combustion engine,
- said turbine comprises a jacket provided at least in part in a body thereof, arranged for the passage of a cooling fluid and in hydraulic communication with an inlet channel and an outlet channel hydraulically connected to a cooling circuit of said motor unit of said internal combustion engine,

- said inlet channel, outlet channel and lubrication channel being integrated in said turbocharger assembly in correspondence of a connection interface between said turbocharger assembly and said motor unit of said internal combustion engine.

Brief description of the drawings

The invention will now be described with reference to the attached drawings, wherein:

- figure 1 is a perspective view of an internal combustion engine according to the present invention,
- figure 1A is a view according to arrow I of figure 1,
- figure 2 is a view according to arrow II of figure 1,
- figure 3 is a perspective view corresponding to figure 1 but with some components removed for the sake of clarity,
- figure 4 is a view corresponding to figure 2 with further components removed,
- figure 5 is a sectional view according to line V-V of figure 4,
- figure 6 is a sectional view according to line VI-VI of figure 3,
- figure 7 is a view according to arrow VII of figure 4, and
- figure 8 is a perspective view of a first variant of the internal combustion engine according to the invention.
- figure 8A is a view according to arrow VIII of figure 8,
- figure 9 is a view according to arrow IX of figure 8,
- figure 10 is a partly sectioned view according to line X-X of figure 8,
- figure 11 is a view corresponding to figure 8

but with some components removed for the sake of clarity

- figure 12 is a perspective view of a second variant of the internal combustion engine according to
5 the present invention

- figure 13 is a perspective view according to arrow XIII of figure 12,

figure 14 is a view according to arrow XIV of figure 12,

10 - figure 15 is a view corresponding to figure 12 but with some components removed,

- figure 15A is a schematic view according to arrow XV of figure 12,

15 - figure 16 is a perspective view partly sectioned according to line XVI-XVI of figure 12,

- figure 17 is a view corresponding to figure 16 but with some components removed for the sake of clarity, and

20 - figure 18 is a sectional view according to line XVIII-XVIII of figure 15.

Detailed description of the invention

An internal combustion engine according to the present invention is indicated with 1 in figures 1, 1A, 2, 3. The internal combustion engine 1 comprises a
25 motor unit 1A, in turn including a cylinder block 2 and a head 3, and a turbocharger assembly 4. The turbocharger assembly 4 comprises a turbine 5, a compressor 6 and a central body 7 interposed between the turbine 5 and the compressor 6. The turbine 5
30 comprises a body 8 including a first and a second portion 9, 10. In the present description the body 8 will at times also be indicated using the term "turbine body", with an entirely equivalent meaning.

An inlet port (not visible) in fluid dynamic
35 communication with a volute 11 and with an outlet port

12A of said turbine, are provided in said first and second portion 9, 10. A boost pressure control valve (so-called "wastegate valve") is arranged in the turbine body 8 and it is actuated by means of an actuator 12B, preferably of the pneumatic type.

Analogously, the compressor 6 comprises a body 13 including a first portion 14 and a second portion 15 and further comprises, provided in said first and second portion 14, 15, an intake port 16, a volute 16A and an outlet port 17. A bypass valve 17A (so-called "dump valve") of the compressor 6 is fixed on the portion 15, in correspondence of the outlet port 17. In the present description the body 13 will also at times be indicated using the term "compressor body", with an entirely equivalent meaning.

The central body 7 comprises a first and a second portion 18, 19 and a hole 19A provided on the second portion 19. The first portions 9, 14, 18 are integral to each other and integral with the head 3. The second portions 10, 19, 15 are integral to each other and they are separable from the head 3. Thus, the turbocharger assembly 4 substantially comprises a first and a second semi-shell A, B put close together along a separation plane C, preferably with the interposition of a gasket (or, alternatively a sealing paste), and fixed to each other by means of threaded joints 20. In particular, the first semi-shell A is defined by the first portions 9, 14, 18 respectively of the turbine body 8, of the compressor body 13 and of the central body 7, while the second semi-shell B, separable from the first semi-shell A, is defined by the second portions 10, 15, 19 still, respectively, of the turbine body 8, of the compressor body 13 and of the central body 7.

The joining of the two semi-shells A, B substantially provides a stator assembly of the

turbocharger assembly 4, i.e. an assembly of fixed parts. Thus the turbine body 8, the compressor body 13 and the central body 7 are part of the stator assembly.

Referring to figure 3, a rotor assembly 21 -
5 comprising a first impeller 22 rotatably mounted inside the turbine body 8, a bearing pack 23 housed in a cavity 24 of the central body 7 - is installed in the turbocharger assembly 4. A lubrication channel 24A arranged for the passage of a pressurised lubricating
10 fluid and hydraulically connected with a lubrication circuit of the engine 1 ends up inside the cavity 24 in correspondence of a hole 24B.

The rotor assembly further comprises a second impeller 25 rotatably mounted inside the body 13 of the
15 compressor 6. The first and the second impeller 22, 25 are coaxial to each other and with respect to a rotation axis X (around which they are also rotatable) and they are rotatably connected to each other by means of a shaft 26.

Referring to figures 3, 4, 6, in this embodiment
20 the bearing pack 23 comprises a sleeve 27 having a radial hole 27A and a plurality of annular grooves 28, 29, 30 in hydraulic communication, by means of respective radial channels 31A, 31B, 31C, with an
25 internal cavity 32 provided in the sleeve 34. When the rotor assembly 21 is inserted into the cavity 24, the groove 29 is also positioned in correspondence with the hole 24B and the radial hole 27A is located in correspondence with the hole 19A.

30 The shaft 26 which is rotatably supported by a bushing 33 is inserted into the internal cavity 32. In other embodiments, the rotor assembly 21 comprises a bearing pack of the rolling type. In any case, generally there are many other options for providing
35 the bearing pack 23.

A jacket 34 arranged for the passage of a cooling fluid is provided in the body 8 of the turbine 5 and partly in the central body 7. Furthermore, a collection channel 35 arranged for the collection of lubricating fluid and in direct hydraulic communication with the grooves 28, 30 are provided in the central body 7.

The jacket 34 and the channel 35 are provided partly in semi-shell A, partly in semi-shell B. Each portion of the jacket 34 and of the channel 35 is hydraulically connected to the respective portion provided on the second shell B when the first and the second shell A, B are joined together by means of the threaded joints 20.

Firstly, referring to figure 5 and also to figures 3, 4, the head 3 comprises an exhaust manifold 36 fluid dynamically connected to exhaust conduits 37 associated to each combustion chamber 38 provided in the head 3. The exhaust manifold 36 is cast integrally with the head 3 and it is surrounded by a cooling fluid circulating in a first and a second cooling jacket 39, 40, in hydraulic communication with a cooling circuit of the motor unit 1A of the internal combustion engine 1.

The jacket 34 provided within the body 8 of the turbine 5 and partly within the central body 7 is hydraulically connected, by means of an inlet channel 41 and an outlet channel 42 respectively to the first and to the second cooling jacket 39, 40.

The inlet and outlet channels 41, 42 are integrated in the turbocharger assembly 4, particularly in the stator assembly, in correspondence of a connection interface between the turbocharger assembly 4 itself and the internal combustion engine 1.

In the present description, the term "connection interface" is used to indicate a physical or geometric

surface (ideal) along which the turbocharger assembly 4 and the motor unit 1A of the internal combustion engine 1, particularly the stator assembly of the turbocharger assembly 4 are joined to the head 3 of the motor unit 1A.

Thus, in the embodiment described herein, the connection interface between the turbocharger assembly 4 and the head 3 is a surface which ideally separates the semi-shell A from the head 3 though they are integrally cast together. Analogously, the channel 24A hydraulically connected to a lubrication circuit of the motor unit 1A is integrated in the turbocharger 4, particularly in the stator assembly, in correspondence of the abovementioned connection interface.

In other words, assuming to ideally separate the stator assembly from the head 3 (or, generally from the motor unit 1A) along the abovementioned connection interface (thus obtaining two complementary separation surfaces, one on the stator assembly and one on the head), the result would be a stator assembly having a series of terminal ports ending up on the separation surface thereof and corresponding, respectively, to channels 41, 42 and to channels 24A, 35 on one hand, and a head (preferably) which has, on the separation surface thereof, terminal ports corresponding to the jackets 39, 40 and to two channels hydraulically connected to the lubrication circuit of the motor unit 1A, wherein the terminal ports of the stator assembly and of the head would evidently be in homologous positions (the association is between the channels 41, 42 and , respectively, the jackets 39, 40 and between the channels 24A, 53 with the channels connected to the lubrication circuit of the motor unit 1A of the engine 1, whose type will be addressed further ahead). The interface between the two separation surfaces is

evidently the abovementioned "connection interface".

Furthermore, a sleeve 43 having an internal channel 44 and a fastening flange 45 rigidly connected to the body 13 of the compressor 6 by means of threaded joints 46 is mounted in correspondence of the inlet port 16. It should be observed that the diameter of the channel 44 defines the geometric passage section in the inlet port 16. The sleeve 43 is inserted into a hole 47 provided on the body 13 of the compressor 6 in correspondence of the inlet port 16.

The internal combustion engine 1 operates as follows.

The exhaust gases which are generated following the combustion of the air/fuel mixture inside the combustion chambers 38 traverse the exhaust conduits 37 and are collected by the exhaust manifold 36, which is fluid dynamically connected with the inlet port of the turbine 5.

Thus, the exhaust gases flowing out into the volute 11 and flowing out from the outlet port 12 rotate the first impeller 22 which thanks to the shaft 26 brings the second impeller 25 in rotation. The latter intakes air through the inlet port 16, particularly through the channel 44 and conveys it, compressing it, into the body 13 of the compressor 6 wherefrom, through the outlet port 17, it is delivered to the intake of the internal combustion engine 1, typically after an intercooling step.

The head 3 is preferably made of an aluminium alloy as well as the shells A, B. In order to avoid the melting of the stator assembly, particularly the turbine body 8, due to the high temperatures reached by the exhaust gases, both in the case of compression ignition engines and in the case of spark ignition engines (for which the exhaust temperatures are

definitely higher), a cooling fluid coming from the internal combustion engine 1 passes from the jacket 39 through the inlet channel 41 and flows into the jacket 34 within the turbine body 8 and within part of the central body 7.

Thus, the water traverses the jacket 34 and cools the body 8 and the central body 7 flowing out at a higher temperature through the outlet channel 42 and then returns to the cooling circuit of the motor unit 1A through the jacket 40.

Simultaneously, the lubrication channel 24A, in hydraulic communication with the lubrication circuit of the motor unit 1A, contains lubricating fluid, particularly lubricating oil, which through the groove 29 and the corresponding channel 31B lubricates the internal cavity 41 and the bushing 42, allowing high rotation speeds of the shaft 26 and of the impellers 22, 25.

The lubricating oil is then forced towards the radial channels 31A, 31C and therefrom towards the annular grooves 28, 30 from which it is collected within the channel 35 for recirculation within the lubrication circuit. The channel 35 is in fact hydraulically connected to a return line of the lubricant which propagates from the head 3 towards the cylinder block 2. It should be observed that the heating of the oil is prevented by the cooling fluid which circulates in the jacket 34 and which lowers the temperature of the turbine body 8 and also of the central body 7.

The internal combustion engine 1 according to the invention has a considerable number of advantages with respect to the internal combustion engines of the known type.

In the first place, the pipings, the fittings and

the gaskets required for the hydraulic connection between the channels for the lubricating fluid and the cooling fluid normally located on the central body of the turbochargers of supercharged engines of the known type are eliminated. Furthermore, the number of components that form the turbocharger assembly 5 is considerably lower with respect to the internal combustion engines of the known type, given that the turbocharger assembly 5 substantially comprises the first and the second shell A, B, the rotor assembly 23 and the sleeve 43. This, combined with the elimination of the pipings, the fittings and the gaskets allows considerably reducing the production costs of the turbocharger assembly 5 and of the entire internal combustion engine 1.

However, given that the shells A, B are preferably made of aluminium, it is also possible to obtain considerable saving on the type of material. Actually, by using the jacket 34 for cooling the turbine body 8 (and the central body 7), it is possible to avoid using the expensive Nimonic® superalloys to the advantage of using aluminium which has a definitely much lower cost.

In addition to what has been described above, it should also be observed that the internal combustion engine 1 according to the invention also has several indisputable advantages in terms of the mechanical machining operations on the turbocharger assembly 4. Actually, in order to obtain mechanical machining operations that are accurate and capable of ensuring the dimensional tolerances required for such application, a preferred machining method for the turbocharger assembly 4 is the following.

The head 3 and the shell A, provided in a single piece, are temporarily joined to the shell B, provided separately. In this step, neither the sleeve 43 nor the

rotor assembly 21 are assembled.

The internal cavity 24 of the central body 7 is thus machined, for example by means of various types of tools (for example boring tools or the like), to obtain
5 the combination of surface finishing and geometric tolerances required for the insertion of the rotor assembly 21.

Then, after the temporary separation of the two shells A, B for the removal of chips possibly
10 accumulated following the mechanical finishing, the two semi-shells are put close together once again and fastened by means of threaded joints 20, the rotor assembly 21 pre-assembled as shown in figure 3 is inserted axially along the axis X through the hole 47.
15 In particular, the diameter of the hole 47 must be sufficiently wide to allow the insertion of the rotor assembly 21 inside the cavity 24 is ensured by inserting a pin or a threaded dowel into the holes 19A and 27A, which,
20 as described previously, are aligned when the rotor assembly is inserted into the cavity 24.

Lastly, the sleeve 43 is fitted into the hole 47 and it is fastened to the turbocharger assembly 4 by means of the threaded joints 46.

25 Referring to figures 8, 8A, 9, an advantageous variant of an internal combustion engine according to the invention is indicated with 100. The components identical to those described previously regarding the internal combustion engine 1 are indicated using the
30 same reference number. The internal combustion engine 100 comprises a motor unit 101, in turn including the cylinder block 2 and a head 103, and a turbocharger assembly 104.

The turbocharger assembly 104 comprises a turbine
35 105, a compressor 106 and a central body 107 interposed

therebetween. The turbine 105 and the compressor 106 comprise respective bodies 108, 113 integral with the central body 107. In the present description, the body 108 of the turbine 105 will at times be indicated using
5 the term "turbine body", with an equivalent meaning.

Similarly, the body 113 of the compressor 106 will at times be indicated using the term "compressor body", with an equivalent meaning.

Analogously to the turbocharger assembly 4, the
10 bodies 108, 113 and 107 define a stator assembly of the turbocharger assembly 104.

The bodies 108, 113 and the central body 107 are cast integrally with the head 103. It should be observed that, substantially, this variant differs from
15 the internal combustion engine 1 solely due to the fact that the stator assembly of the turbocharger assembly 104 does not comprise the semi-shells A, B, which - on the contrary - are made in a single piece together and with the head 103.

20 The head 103 is furthermore substantially identical in the structure outside and inside the head 3 - in particular it comprises the exhaust manifold 36 integrally cast therein and fluid dynamically connected to an inlet port of the turbine 105 and the jackets 39,
25 40 - obviously except for the fact that the entire stator assembly of the turbocharger assembly 104 is integral therewith, not a single semi-shell as in the case of the head 3.

The internal structure of the turbocharger
30 assembly 104 (figure 10, 11) is also identical, wherein
- the turbine 105 comprises the volute 11, the outlet port 12, the wastegate valve 12B and the jacket 34, which are in such case provided without interruption in the turbine body 108 and in part of the
35 central body 107,

- the compressor 106 comprises, all provided without interruption, the inlet port 16, the volute 16A, the outlet port 17 and the bypass valve (dump valve) 17A;

5 - the central body 107, alongside a part of the jacket 34, comprises the hole 19A, and the cavity 24 with the lubrication channel 24A and the hole 24B, wherein the cavity 24 accommodates the rotor assembly 21 comprising the bearing pack 23, the first impeller 22 rotatably mounted within the turbine body 108 and
10 the second impeller 25 rotatably mounted within the compressor body 113.

Furthermore, the collection channel 35, also in this case provided without interruption, is arranged in
15 correspondence of the central body 107.

Analogously to the internal combustion engine 1, the sleeve 43 inserted into the hole 47 provided in the compressor body 113 and flanged to the same compressor body 113 by means of the threaded joints 46 is
20 furthermore present.

The method for machining and mounting the turbocharger assembly 104 is substantially identical to that described regarding the turbocharger assembly 4, with the obvious absence of the step of temporarily
25 joining the shells A, B which are in such case made in a single piece together. Analogously to the turbocharger assembly 4, the rotor assembly 21 is axially inserted through the hole 47 in the internal cavity 24, and it is axially blocked by means of a pin
30 or threaded dowel inserted into the holes 19A, 27A.

It should also be observed that the geometry of the turbocharger assembly may vary slightly with respect to what has been illustrated by way of example given that, depending on the process used for the
35 casting of the head 103 and of the turbocharger

assembly 104 (for example die casting or lost foam casting), it may require arranging one or more holes, for supporting the casting cores, subsequently closable by means of plugs.

5 The engine 100 preserves the same advantages described regarding the internal combustion engine 1 and it further amplifies some of them. In particular, the number of components required for assembling the turbocharger assembly 104 is further reduced in this
10 case because the stator assembly is substantially cast integrally with the head 103 and both are preferably made of an aluminium alloy.

Referring to figures 12 to 18, a second variant of a turbocharged internal combustion engine according to
15 the present invention is indicated with 200. The components identical to those described previously regarding the engines 1, 100 are indicated using the same reference number and have an analogous function.

The internal combustion engine 200 comprises a
20 motor unit 201, in turn including the cylinder block 2 and a head 203, and a turbocharger assembly 204. The turbocharger assembly 204 comprises a turbine 205, a compressor 206 and a central body 207 interposed therebetween. The turbine 205 comprises a body 208 (in
25 the present description also indicated, with analogous meaning, using the term "turbine body") including a first connection flange 209 in correspondence of which an inlet port 210 (figure 15A), a volute 211 and an outlet port 212 (figure 16) end up. Analogously to the
30 turbines 5, 105, the turbine 205 comprises a wastegate valve controlled by an actuator, preferably pneumatic, (wastegate valve and respective actuator were removed for the sake of clarity).

Referring to figures 12, 16, the compressor 206
35 comprises a body 213 (in the present description also

indicated, with analogous meaning, using the term "compressor body"), an intake port 216, a volute 216A and an outlet port 217, in correspondence of which, analogously to the compressors 6, 106, a bypass valve
5 (dump valve), herein removed for the sake of clarity, is mounted.

Similarly to the turbocharger assemblies 4, 104, the turbocharger assembly 204 comprises a stator assembly defined by the bodies 208, 213 and by the
10 central body 207. Furthermore, the central body 207, the turbine body 208 and the compressor body 216 are preferably made in a single piece.

Referring to figures 16, 17, within the central body 207 there is housed the rotor assembly 21
15 comprising the first impeller 22 rotatably mounted in the turbine body 208, the bearing pack 23 housed in the cavity 224, in which a lubrication channel 224A ends up by means of a hole 224B. The lubrication channel 24A is integrated in the stator assembly in correspondence of
20 the connection flange 209, wherein it ends up with a terminal port 224C.

The rotor assembly 21 further comprises the second impeller 25, rotatably mounted in the compressor body 213. The first and the second impeller 22, 25 are
25 coaxial to the axis X and rotatably connected by means of the shaft 26.

The further components of the rotor assembly 21 will not be described in detail again because they are identical to those described previously. Furthermore,
30 analogously to what has been described previously, the rotor assembly 21 is arranged for the insertion in axial direction into the turbocharger assembly 204, particularly into the cavity 224.

Referring to figures 15A, 16, 17, within the
35 turbine body 208 and in part within the central body

207 there is provided a jacket 234 in hydraulic communication with an inlet channel 234A and an outlet channel 234B integrated in the stator assembly in correspondence of the connection flange 209 and ending
5 up thereon by means of respective terminal ports 234C, 234D. Furthermore, a collection channel 235 analogous to the channel 35 of the assemblies 4, 104 is provided in the stator assembly of the turbocharger assembly 204. The channel 235 is in part integrated within the
10 connection flange 209 and ends up thereon with a terminal port 235A. Substantially, four terminal ports 224C, 234C, 234D and 235A are provided in correspondence of the connection flange 209.

Referring to figures 15, 18, the head 203
15 comprises an exhaust manifold 236 cast integrally therewith and fluid dynamically connected to exhaust conduits 237 associated to respective combustion chambers 238.

The exhaust manifold 236 is surrounded by a first
20 and a second cooling jacket 239, 240 which end up, by means of terminal ports 239A, 240A, in correspondence of a second connection flange 248. Furthermore, ending up in correspondence of the connection flange 248 are:

- two terminal ports 249, 250 hydraulically
25 connected to the lubrication circuit of the motor unit 201 of the engine 200 and in positions homologous to those of the terminal ports, respectively, 224C and 235A;

- the exhaust manifold 236 by means of an outlet
30 port 251 substantially shaped identically to the inlet port 210 on the flange 209 and in homologous position with respect thereto.

In particular, the terminal port 249 is connected to the pressurized branch of the lubrication circuit of
35 the motor unit 210, while the port 250 is connected to

a return branch of the lubricant, at low pressure.

The turbocharger assembly 204 and the head 203 are coupled by means of mechanical connection of the first and of the second connection flanges 209, 248 (preferably by means of threaded joints 252): in such manner the flanges 209, 248 define a connection interface between the turbocharger assembly and the motor unit 201, in particular the head 203. Furthermore, the mechanical connection of the abovementioned flanges:

- fluid dynamically connects the outlet port 251 to the inlet port 210 of the turbine 205;

- hydraulically connects the terminal ports 234C and 239A and the terminal ports 234D and 240A, hydraulically connecting, in such manner, the jacket 234 to the jackets 239, 240 and to the cooling circuit of the motor unit 201 of the internal combustion engine 200;

- hydraulically connects the terminal ports 224C and 249 and the terminal ports 235A and 250, hydraulically connecting, in such manner, the lubrication channel 224A and the collection channel 235 to the lubrication circuit of the motor unit 201 of the internal combustion engine.

The sleeve 43 is also mounted on the compressor body 213, in correspondence of the inlet port 216, in a manner entirely identical to that described regarding compressors 6, 106 (figures 12-14 and 16-17)

Regarding the operation of the engine 200, the methods through which the two impellers are brought in rotation by means of the exhaust gases are substantially identical with respect to what has been described regarding the engine 1.

In order to prevent the melting of the stator assembly due to the high operating temperatures of the

turbine 205, the cooling fluid circulating in the engine 200 passes from the first cooling jacket 239 to the jacket 234 through the ports 239A, 234C and traverses the jacket 234 lowering the temperature of the turbine body 208 (and of the central body 207). Then, the cooling fluid flows out, at a higher temperature, from the jacket 234 through the ports 234D, 240A, and flows into the second cooling jacket 240, from which the circulation within the head 203 proceeds.

Similarly, the lubricating fluid, particularly oil, conveyed under pressure to the terminal port 249, flows into the channel 224A through the port 224C, lubricates the rotor assembly 21 and it is collected in the collection channel 235 wherefrom, through the ports 235A, 250, it returns towards the cylinder block 2.

Also the engine 200 preserves the advantages described previously, and additionally requires less complexity regarding the casting of the various parts, given that the turbocharger assembly 204 is separated from the head 203.

However, the choice of the material of the turbine body 208 and of the central body 207 is wider, it is not bound to the use of the same material as the head 203, preferably an aluminium alloy. Actually, an aluminium alloy as well as gray cast iron may be used, hence avoiding expensive Nimonic® steels in any case.

Naturally, the details and embodiments may vary, even significantly, with respect to what has been described and illustrated purely by way of example, without departing from the scope of the present invention as defined by the attached.

CLAIMS

1. A supercharged internal combustion engine (1, 100) comprising:

5 - a motor unit (1A, 101) including a head (3, 103) and an exhaust manifold (36),

- a turbocharger assembly (4, 104) fluid dynamically connected to said exhaust manifold (36,), wherein said turbocharger assembly (4, 104) includes a turbine (5, 105), a central body (7, 107) and a compressor (6, 106),

wherein:

- said turbocharger assembly (4, 104) comprises a lubrication channel (24A) for the passage of a lubricating fluid hydraulically connected to a lubrication circuit of said motor unit (1A, 101) of said internal combustion engine (1, 100),

- said turbine (5, 105) comprises a jacket (34), provided at least in part in a body (8, 108) thereof, arranged for the passage of a cooling fluid and in hydraulic communication with an inlet channel (41) and an outlet channel (42) hydraulically connected to a cooling circuit (39, 40) of said motor unit (1A, 101) of said internal combustion engine (1, 100),

- said inlet channel (41), outlet channel (42) and lubrication channel (24A) being integrated in said turbocharger assembly (4, 104) in correspondence of a connection portion between said turbocharger assembly (4, 104) and said motor unit (1A, 101) of said internal combustion engine (1, 100)

30 - said turbocharger assembly comprises a stator assembly (A, B; 8, 7 13; 108, 107, 113) including the body (8, 108) of said turbine (5, 105), said central body (7, 107) and a body (13, 113) of said compressor (6, 106),

35 - said central body (7, 107) comprises a cavity

(24) wherein a rotor assembly (21) of said turbocharger assembly (4, 104) is housed,

- said rotor assembly (21) comprises a first impeller (22) rotatably mounted within the body (8, 108) of said turbine (5, 105), a bearing pack (23) and a second impeller (25) rotatably mounted in the body (13, 113) of said compressor (6, 106), wherein said first and said second impellers (22, 25) are connected in rotation by means of a shaft (26),

10 - said head (3, 103) comprises an exhaust manifold (36) integrally cast therein (3, 103),

the internal combustion engine (1, 100) being characterized in that:

- at least part (A; 9, 14, 18) of said stator assembly (A, B; 8, 7, 13; 108, 107, 113) is integral with said head (3, 103),

- said bearing pack (23) comprises a sleeve (27) including annular grooves (28, 29, 30) in hydraulic communication, by means of radial channels (31A, 31B, 31C) with an internal cavity (32) provided in said sleeve (27), wherein said shaft (26) is inserted in said internal cavity (32) rotatably supported by a bushing (33).

2. The internal combustion engine (1) according to Claim 1, wherein:

- said stator assembly (A, B; 8, 7, 13) comprises a first semi-shell (A) integral with said head (3) and a second semi-shell (B) separable from said first semi-shell (A),

30 - the body (8) of said turbine (5) comprises a first (9) and a second portion (10), said compressor (6) comprises a body (13) including a first (14) and a second portion (15), said central body comprises a first (18) and a second portion (19),

35 - the first portion (9) of the body (8) of said

turbine (5), the first portion (14) of the body (13) of said compressor (6) and the first portion (18) of the central body (7) are integral with each other and define said first semi-shell (A),

5 - the second portion (10) of the body (8) of said turbine (5), the second portion (15) of the body (13) of said compressor (6) and the second portion (19) of said central body (7) are integral with each other and define said second semi-shell (B),

10 - said jacket (34) and said lubrication channel (24A) are provided in part in said first semi-shell (A) and in part in said second semi-shell (B).

3. The internal combustion engine (100) according to Claim 1, wherein the entire stator assembly (108, 15 107, 113) of said turbocharger assembly (104) is integral with said head (103).

4. The internal combustion engine (1, 100) according to Claim 1, wherein said rotor assembly (21) is arranged for the insertion in axial direction into 20 said turbocharger assembly (4, 104).

5. The internal combustion engine (1, 100) according to Claim 1, wherein said compressor (6, 106) comprises an inlet port (16) in correspondence of which there is mounted a sleeve (43) having an internal 25 channel (44) and a fastening flange (45) rigidly connected to a body (13, 113) of said compressor (6, 106).

FIG. 1

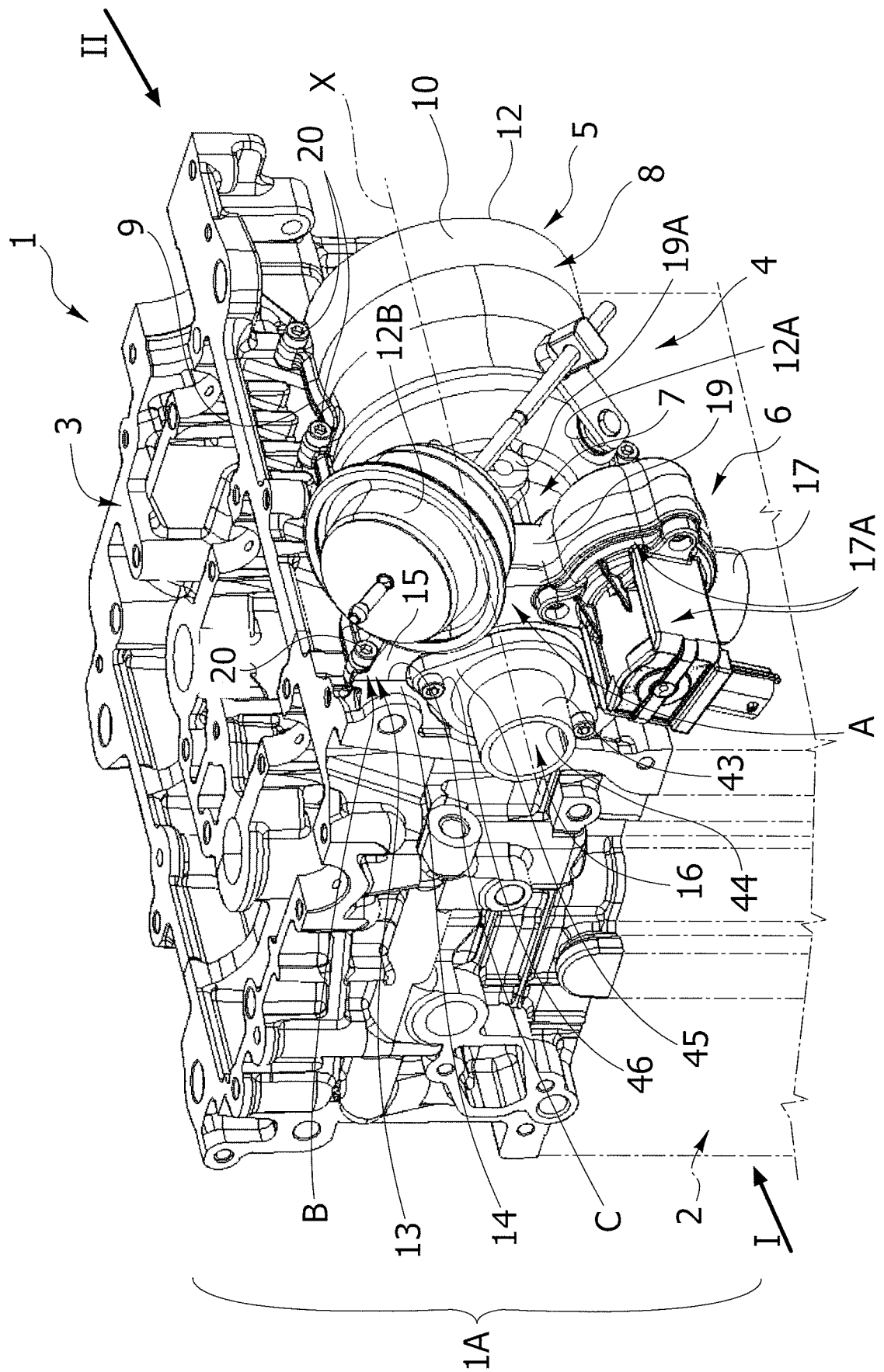


FIG. 2

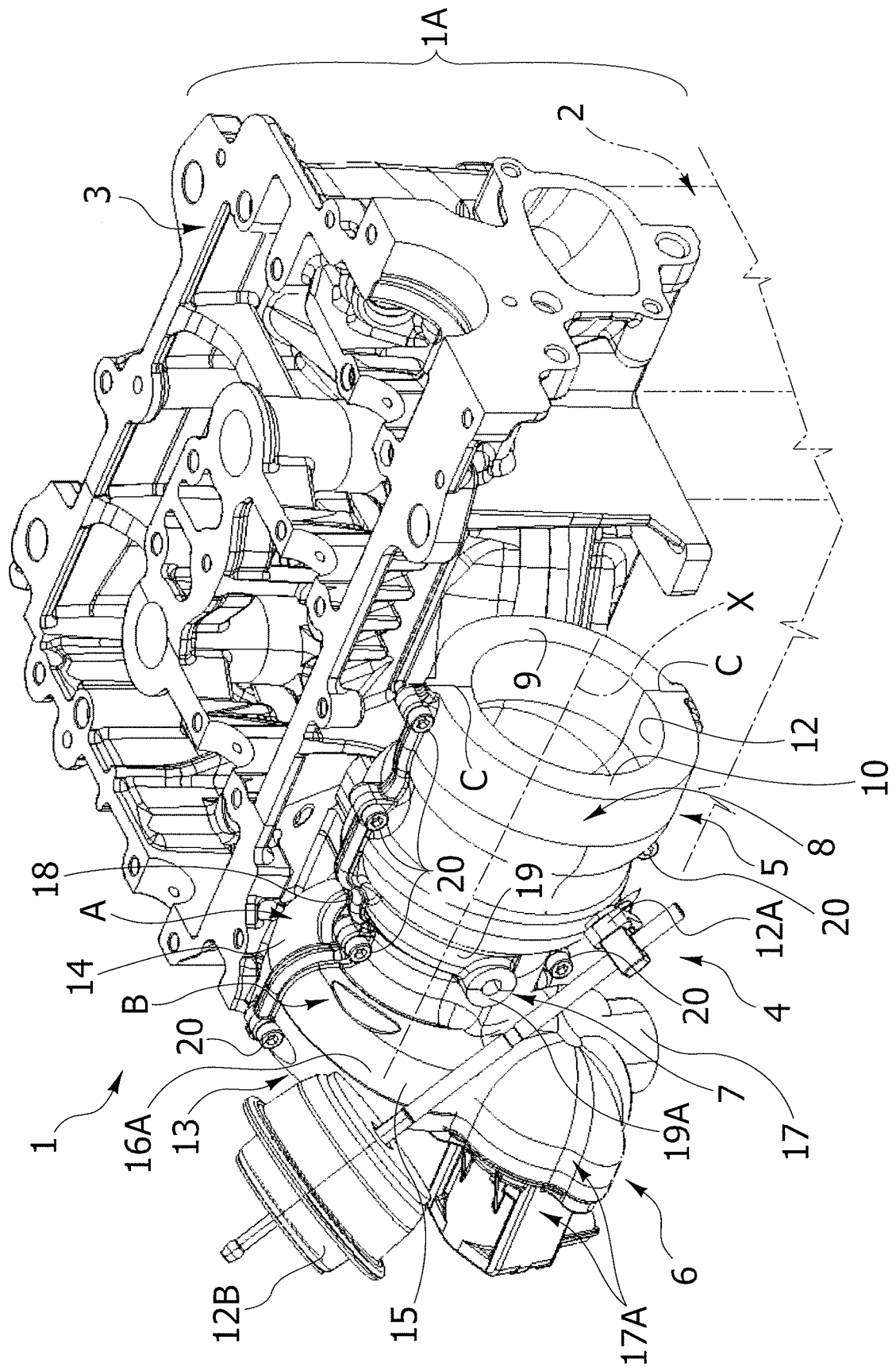


FIG. 3

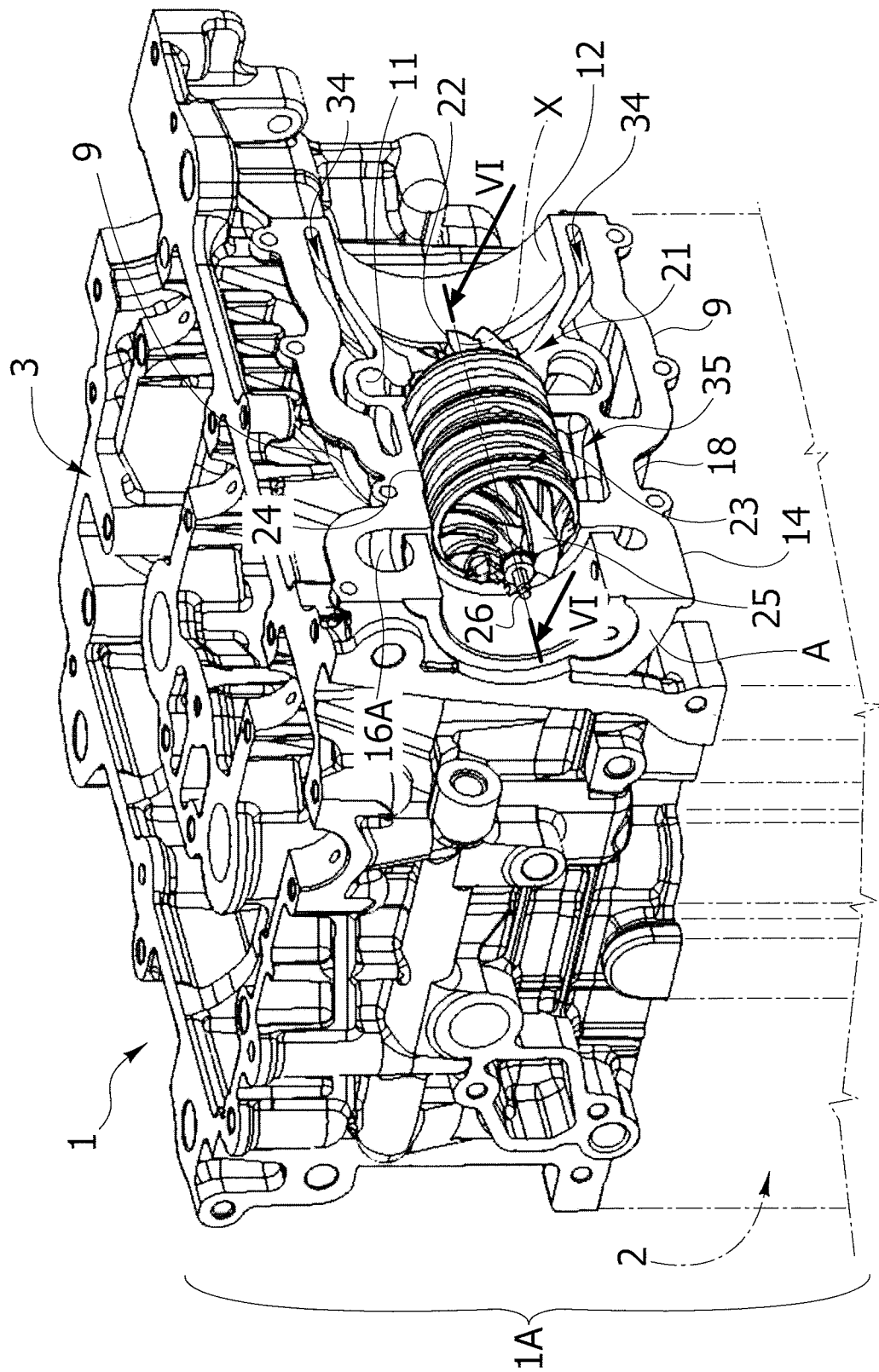


FIG. 4

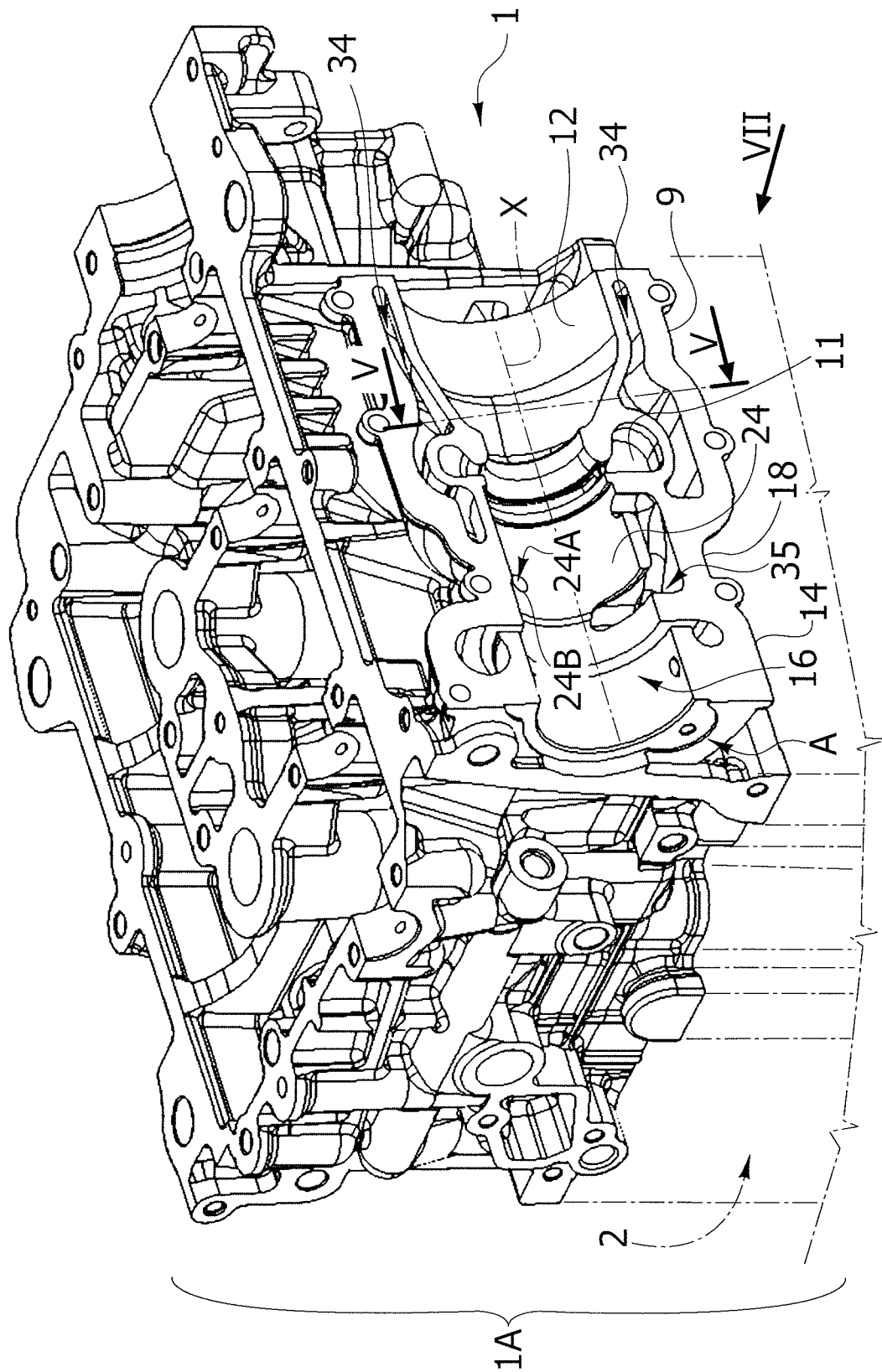


FIG. 5

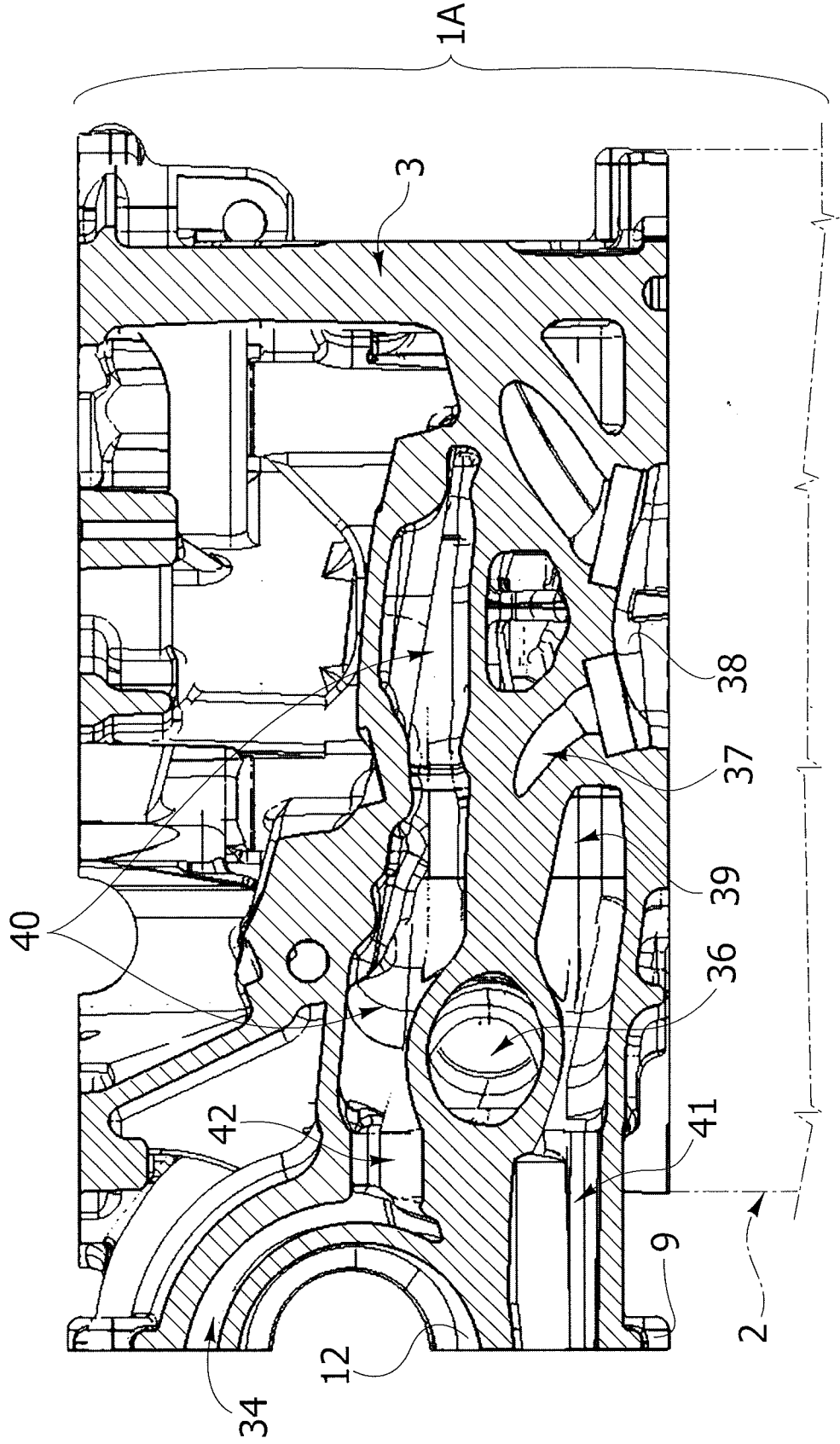


FIG. 6

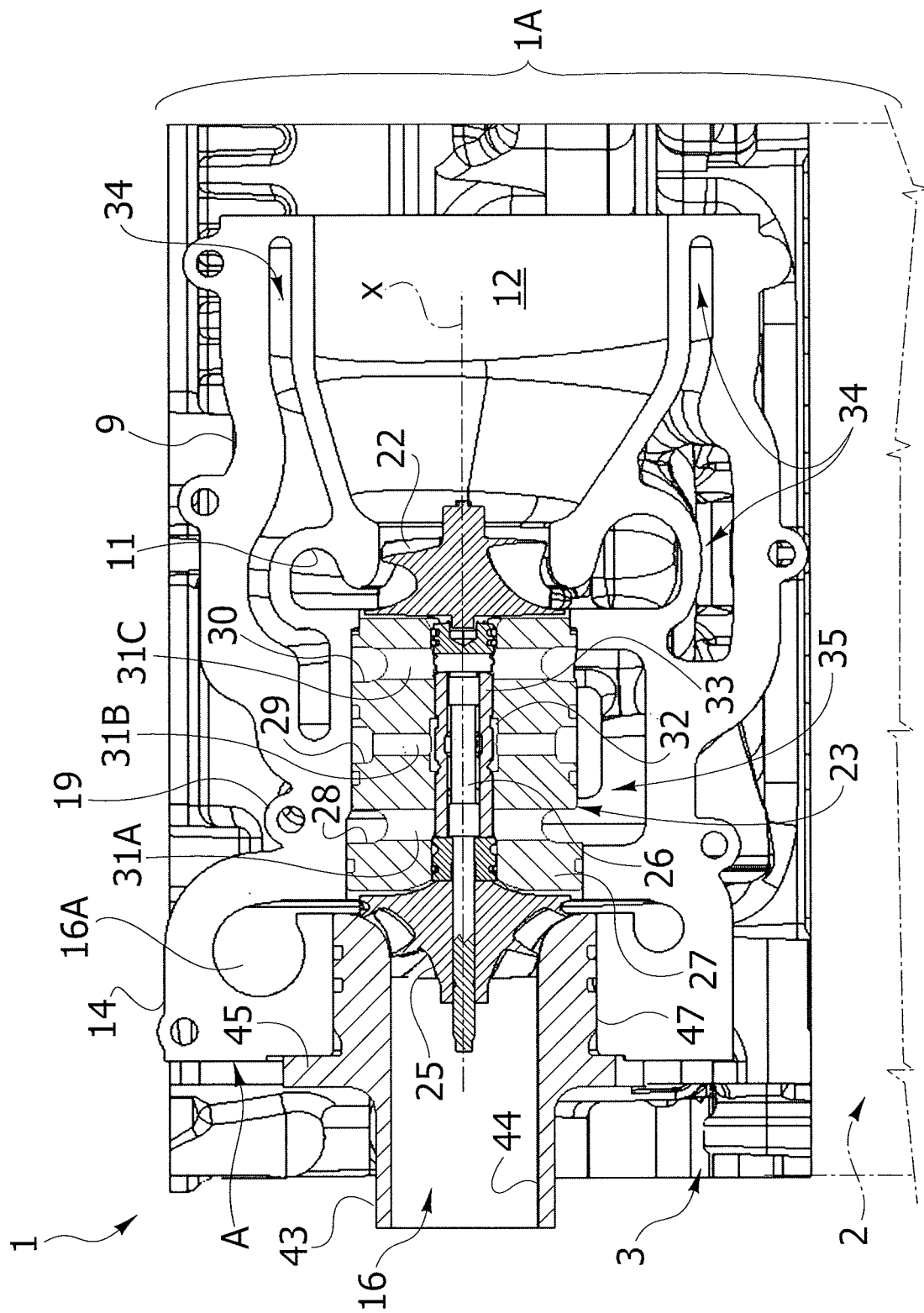


FIG. 7

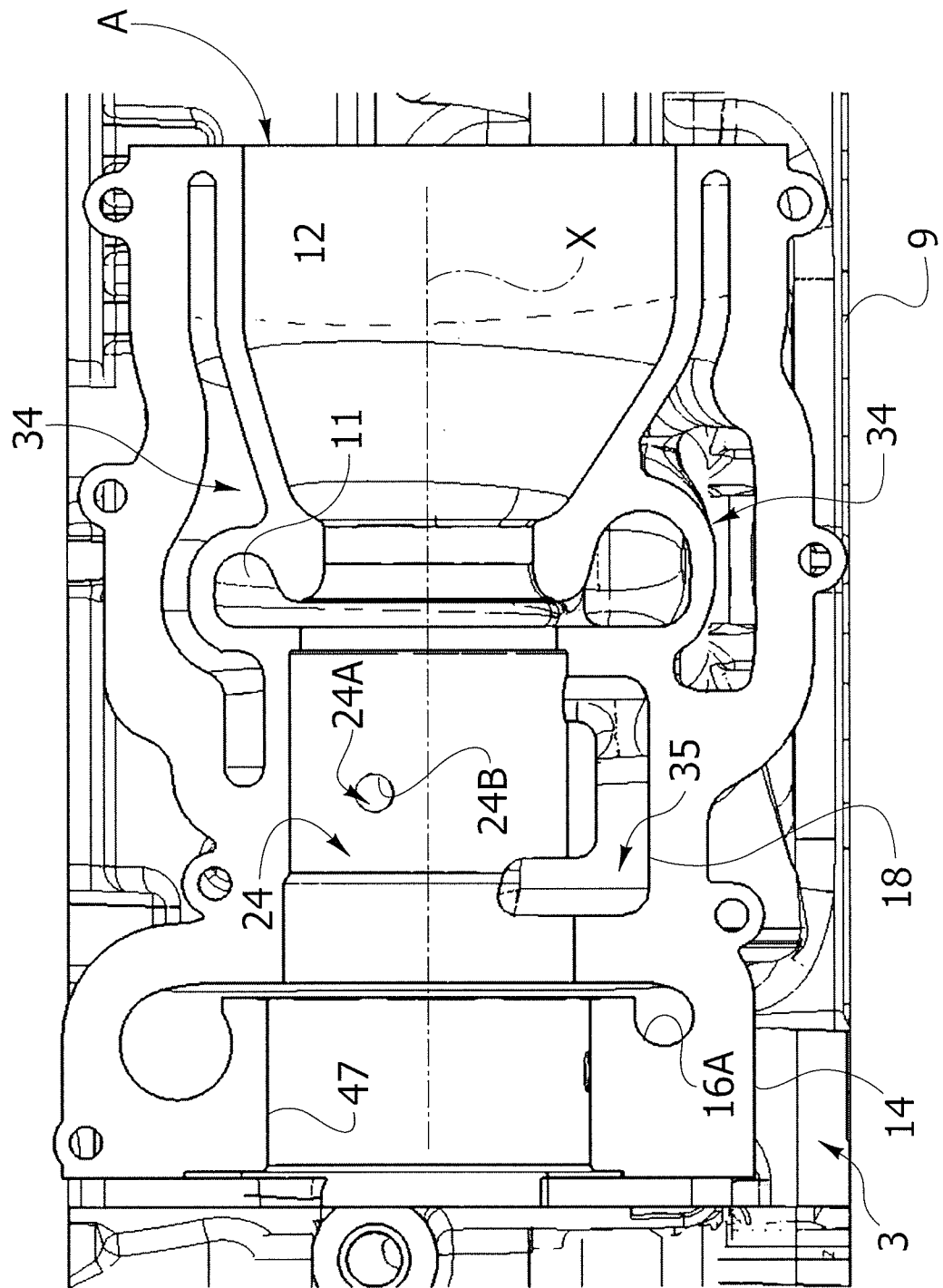
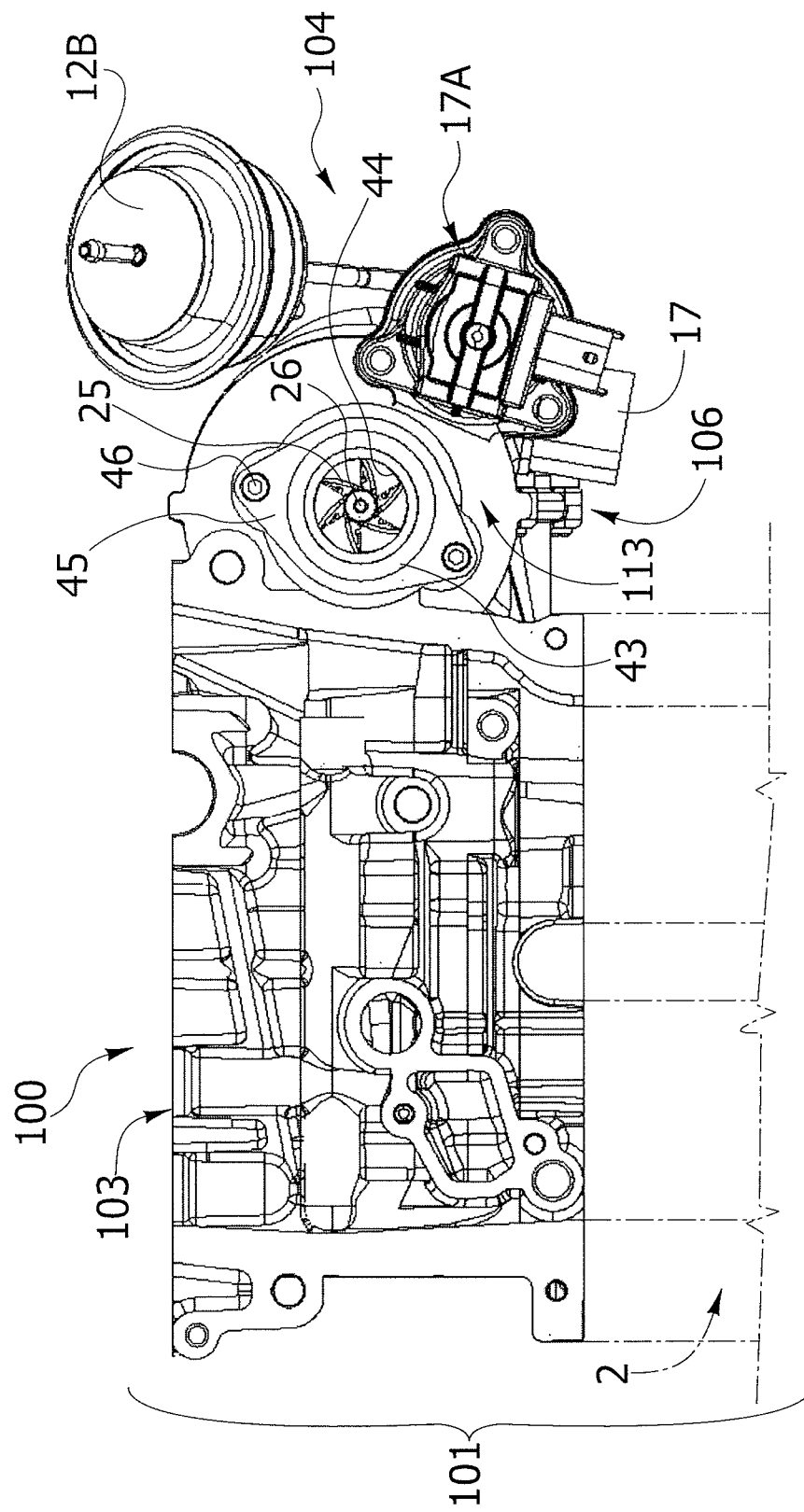


FIG. 8A



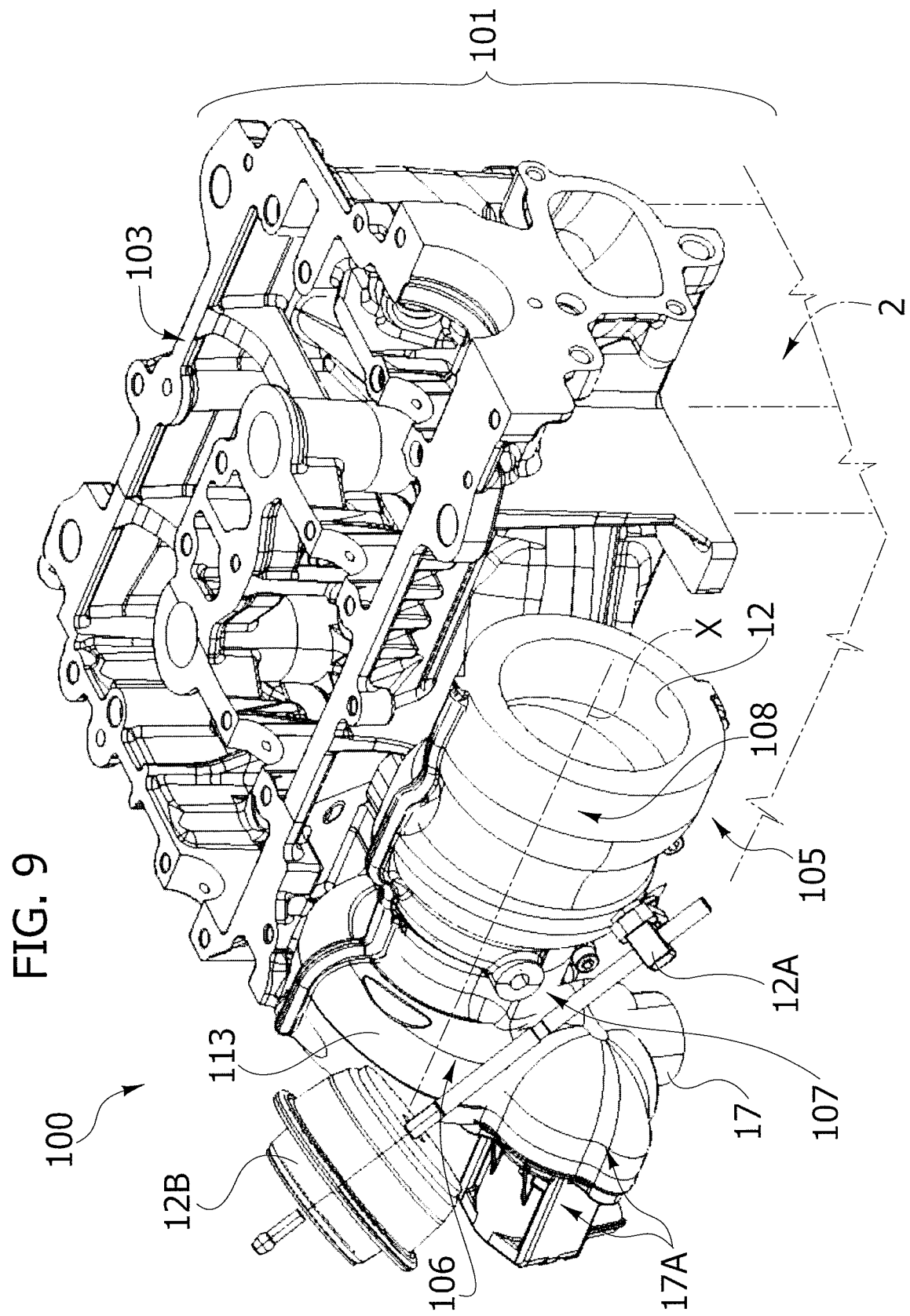


FIG. 10

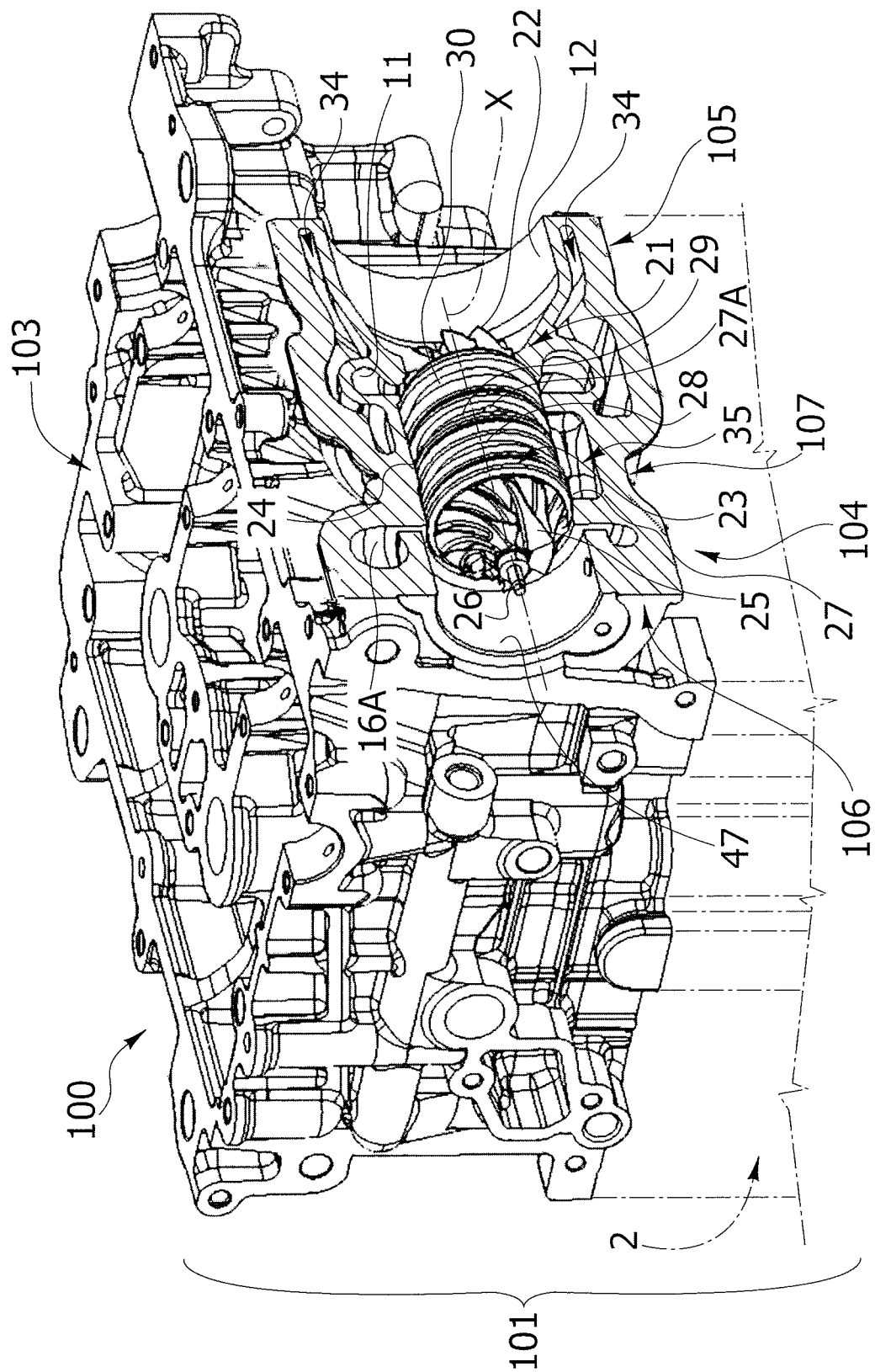


FIG. 11

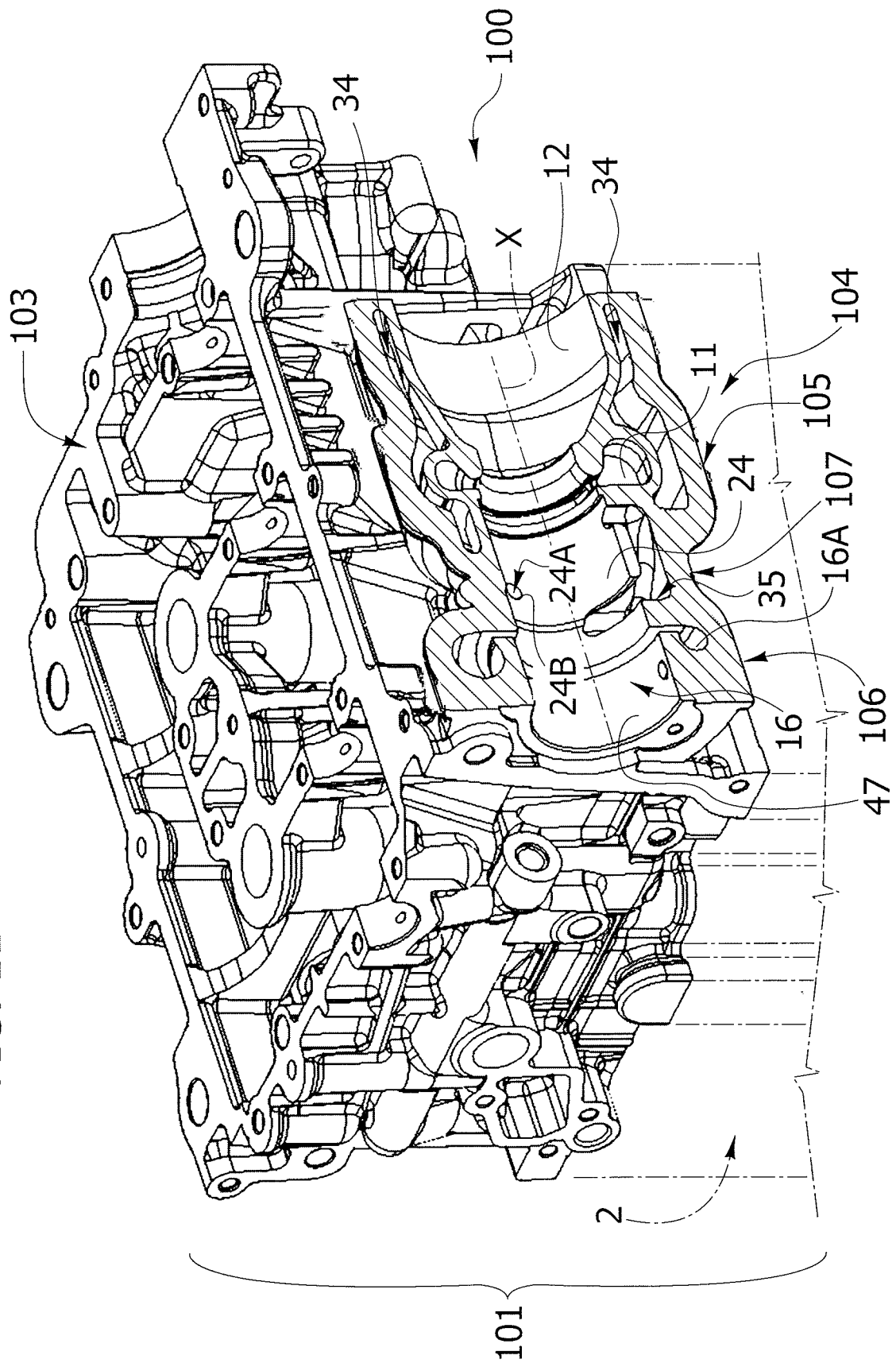


FIG. 12

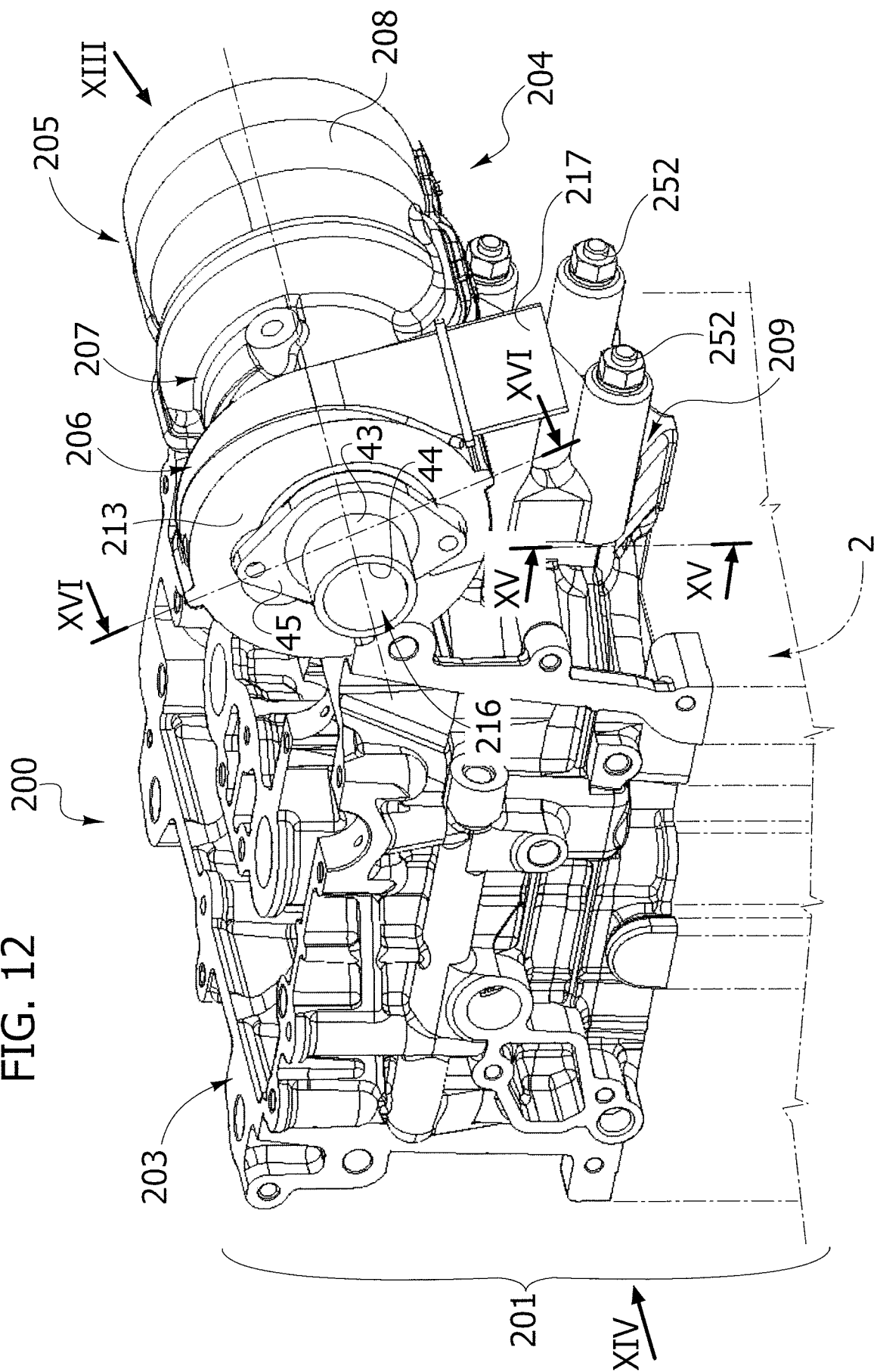
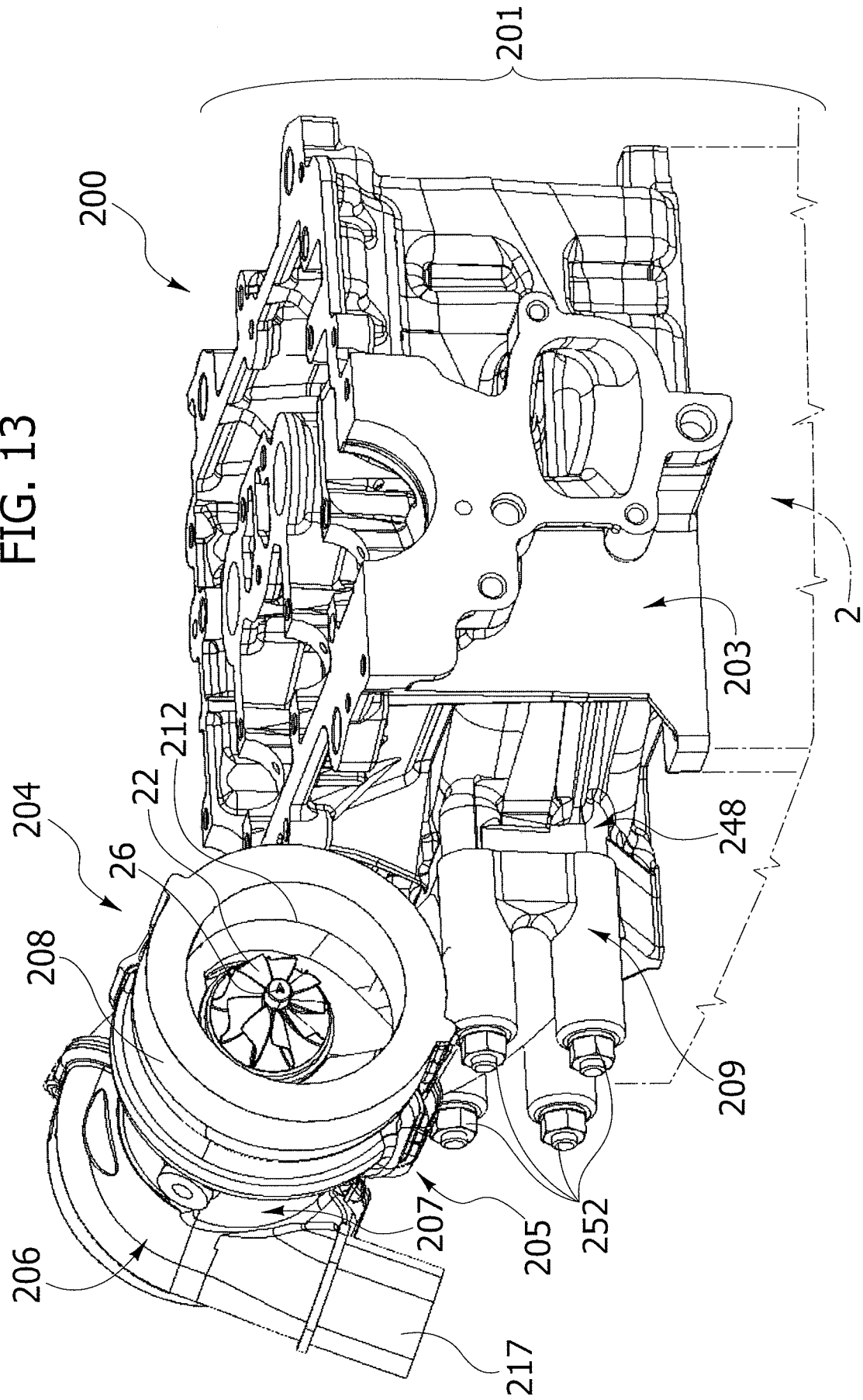


FIG. 13



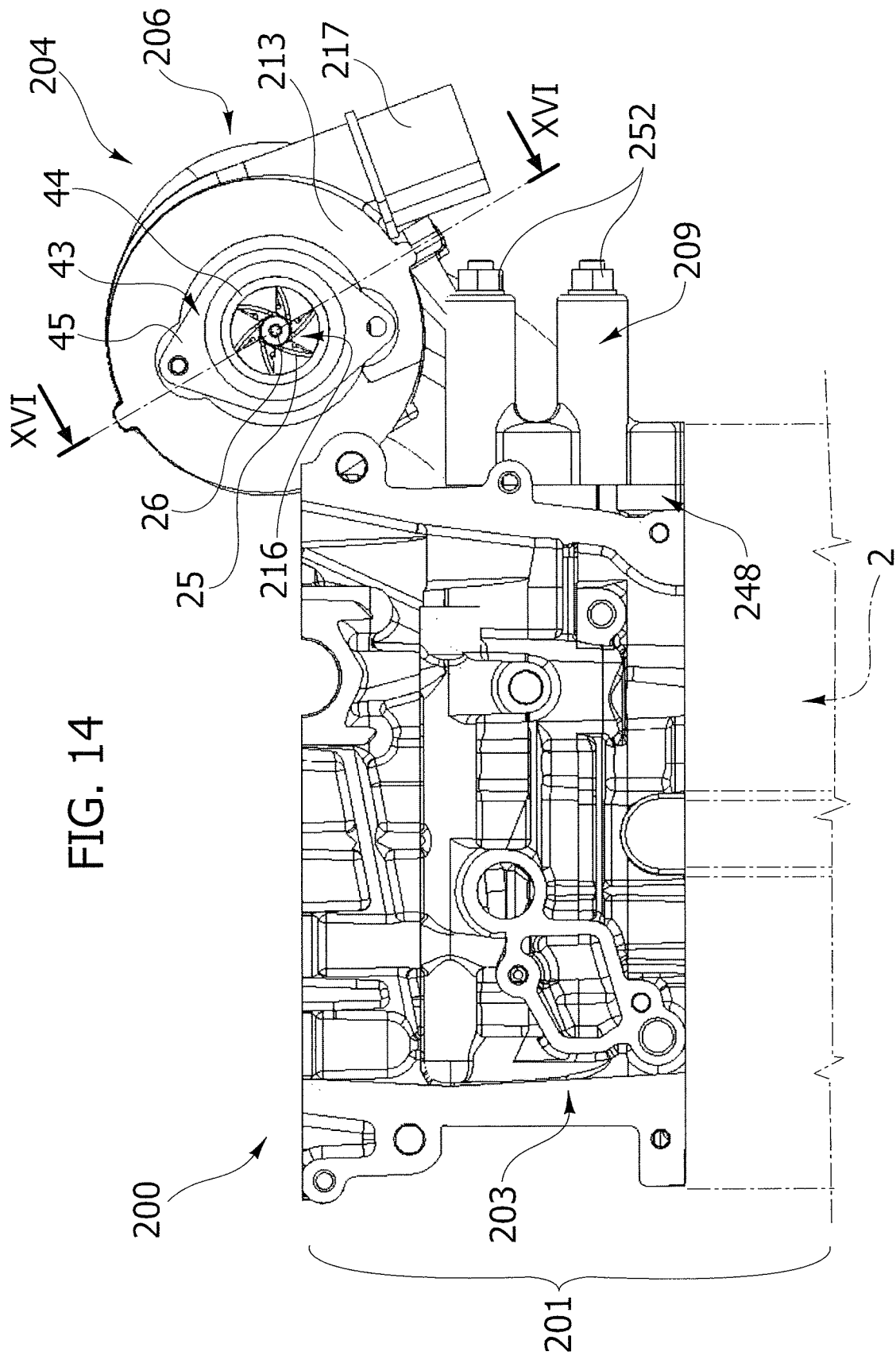


FIG. 15

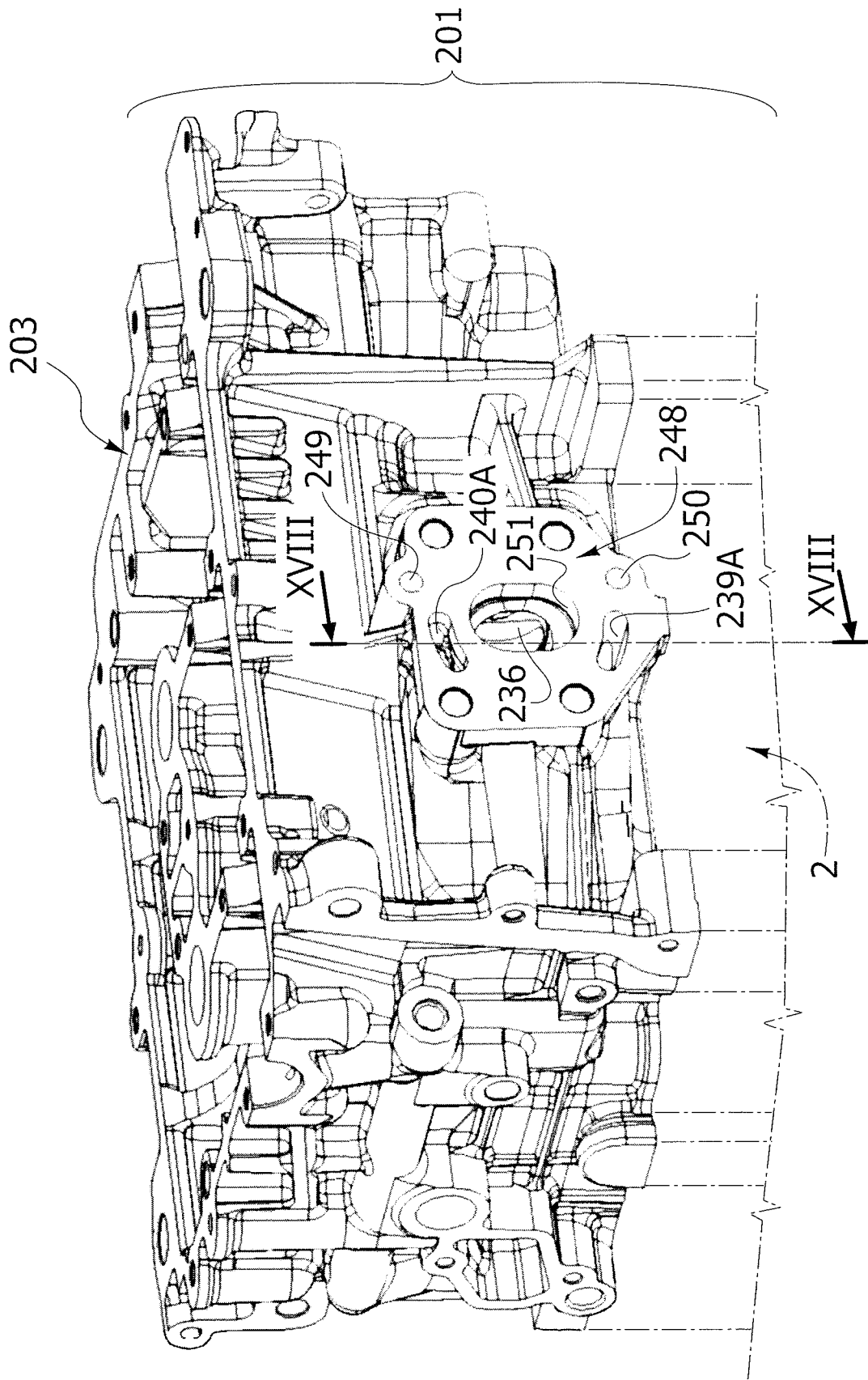


FIG. 15A

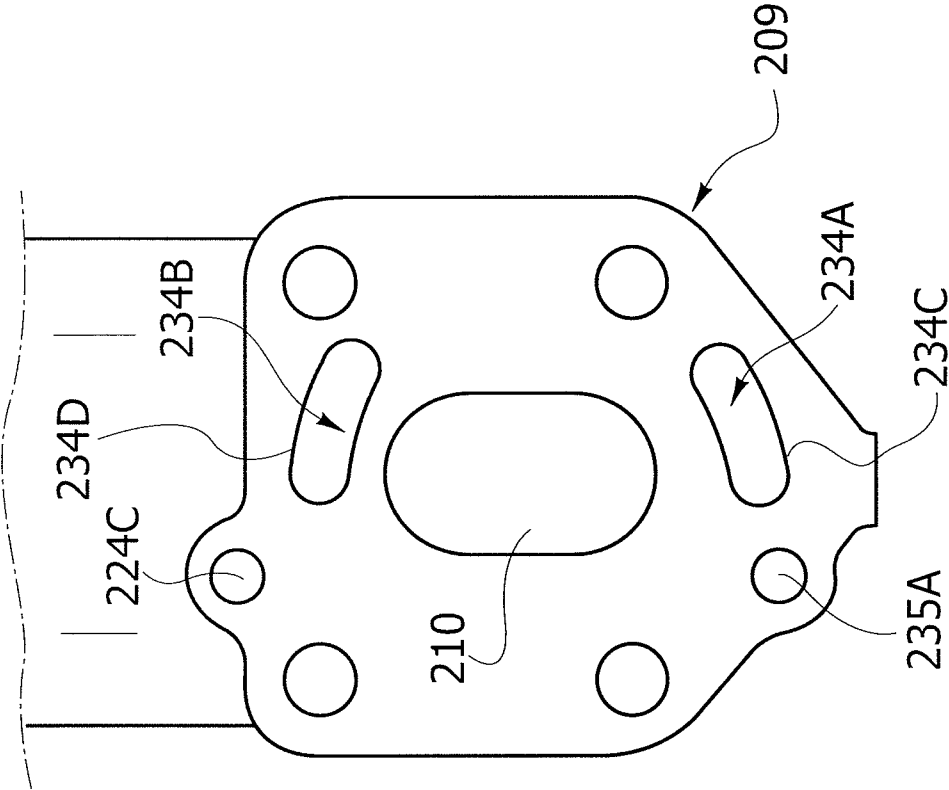


FIG. 16

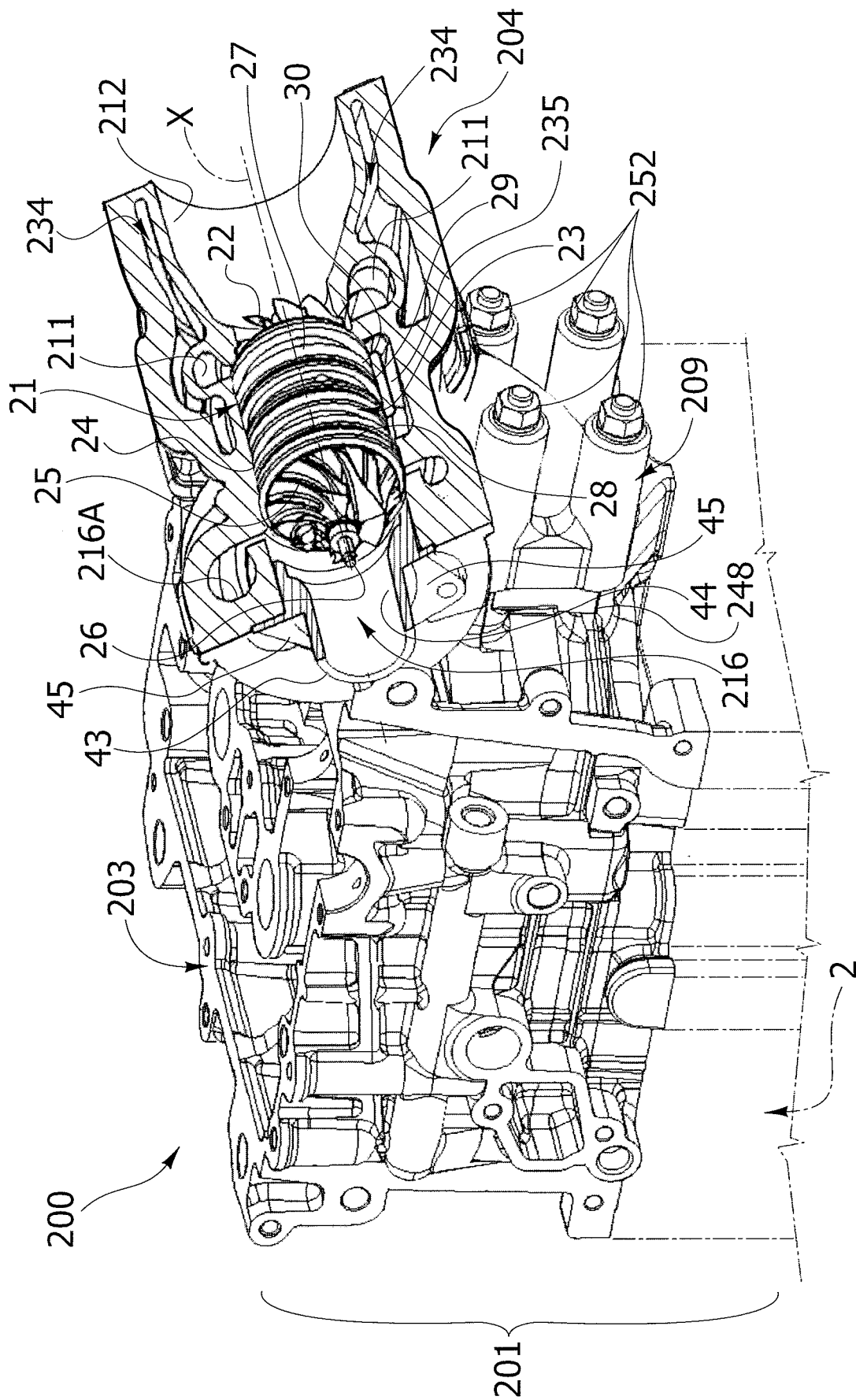


FIG. 17

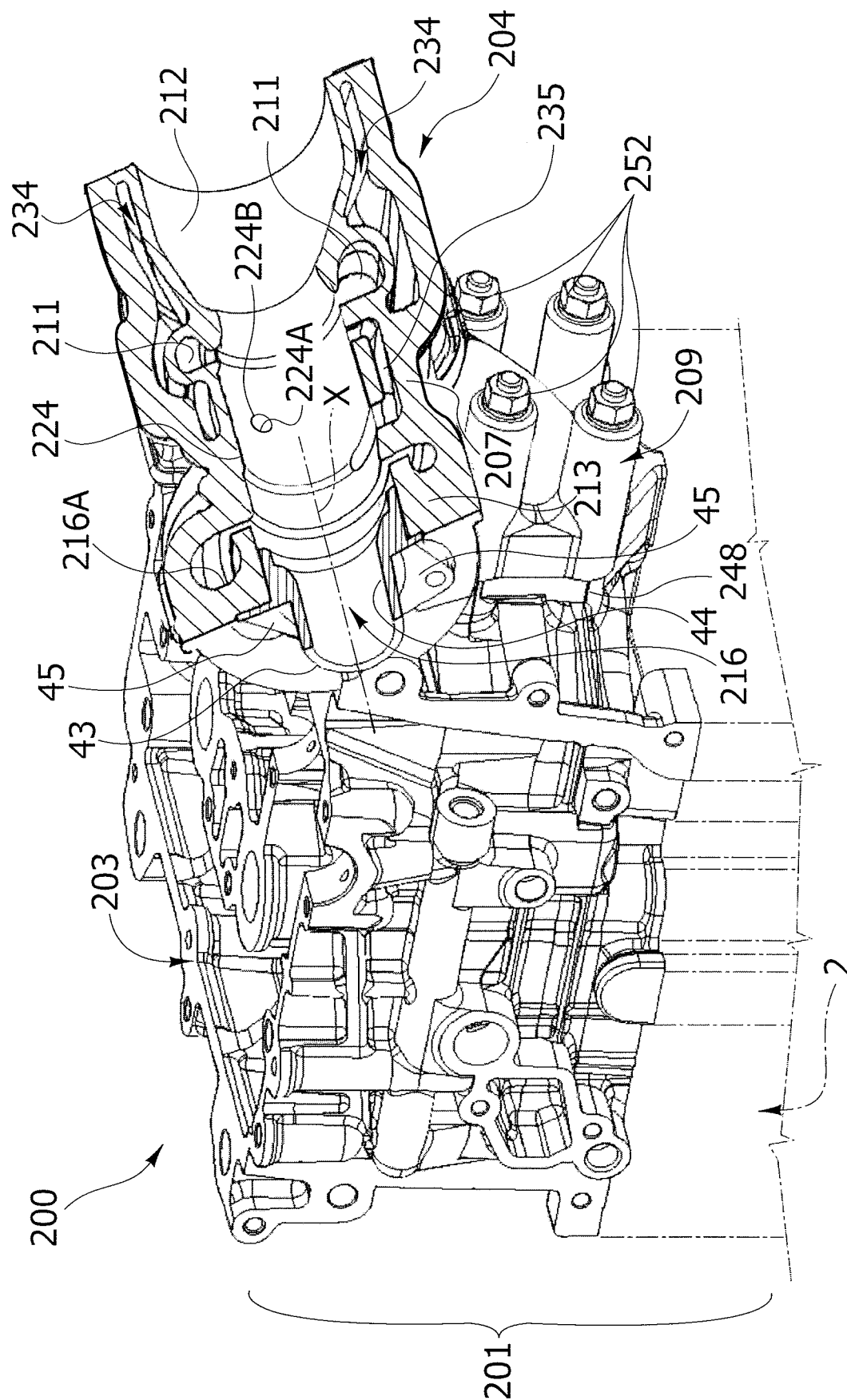


FIG. 18

