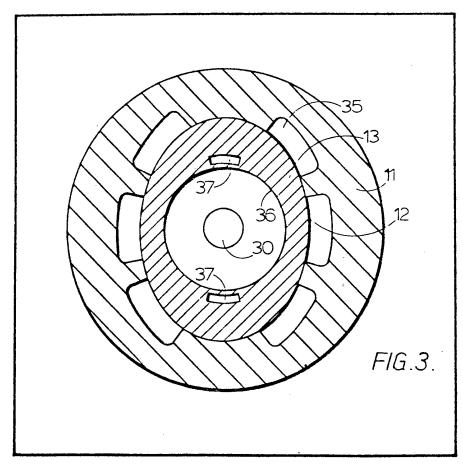
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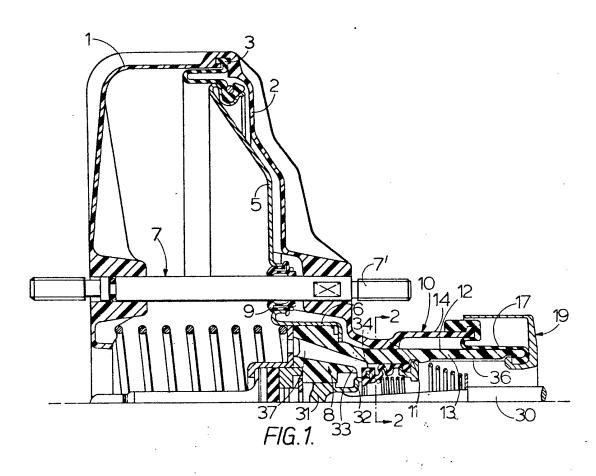
# (54) Servo Boosters for Vehicle Braking Systems

(57) In a servo booster in which the rear part of the valve body 13 is slidably guided in a guide 11 in the rear end of the booster housing, the rear part and the guide have non-

round cross-sections to prevent the valve body from turning relative to the housing. This is of particular benefit when a tie extends through the movable wall for transmitting braking reaction forces. Such a tie has to be sealed to the movable wall by a suitable seal, and any turning of the movable wall might damage the seal.



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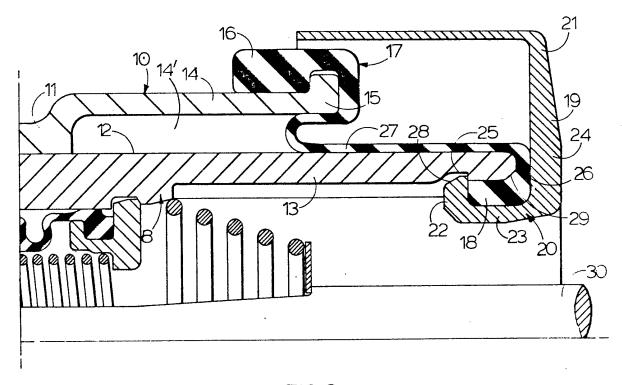
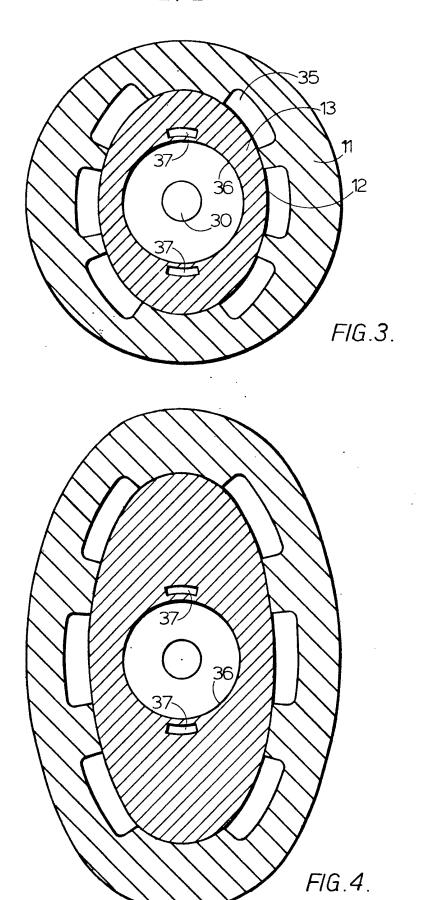


FIG.2.



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## SPECIFICATION Servo Boosters for Vehicle Braking Systems

This invention relates to a servo booster assembly for a vehicle braking system of the kind comprising a housing in which is located a movable wall for augmenting the force applied by a pedal-operated input member to an output member, the movable wall being secured to or integral with a valve body of a valve means controlling the differential pressure applied to the movable wall, the valve body having a part which is guided by guide means in the rear of the housing.

Usually the valve body part is of plain tubular shape and is slidably guided in a rearwardly extending tubular extension of the housing.

According to one aspect of the invention in a servo booster assembly of the kind set forth the guide means has a bore on non-round cross-section, and the outer surface of the valve body part is of non-round cross-section on a plane transverse to the direction of movement of the valve body relative to the housing so as to resist turning of the valve body and movable wall relative to the booster housing.

This aspect of the invention is particularly applicable to servo boosters of the kind disclosed in the Provisional Specifications of our earlier U.K. Patent Application Nos. 43644/77 and 2488/78 Serial No. 1558722. Abandoned.

In the Provisional Specification of our U.K. Patent Application No. 43644/77 Serial No. 1558722 dated 20th October, 1977 we have disclosed a servo booster for a vehicle brake system of the kind having at least two shells defining a housing, a diaphragm assembly comprising a diaphragm of elastomeric or other flexible material dividing the housing into at least two chambers and a support plate, and a control valve assembly, in which the housing is formed by lightweight shells coupled together by means of ties in the form of studs which pass through the diaphragm assembly and which enable said shells to be clamped together.

In the constructions of those Specifications the Patent Application No. 2488/78 abandoned dated 21st January, 1978 we have disclosed a modified construction in which the lightweight housing shells are coupled together by means of at least one stud.

In the constructions of those Specifications the movable wall was proposed to be sealed to each tie by a sliding seal or by a diaphragm. When a booster is provided with one or more ties which extend through the movable wall at positions displaced from the axis of the booster then any turning of the movable wall and control valve assembly relative to the booster housing may result in serious damage to the sliding seal or diaphragm connection with the ties.

Thus, in accordance with a preferable feature of the first aspect of the invention a tie extends through the movable wall from a wall of the housing on one side of the movable wall to a

65 futher wall of the housing on the opposite side of the movable wall, the tie, at least where it passes through the movable wall, having its axis displaced from the longitudinal axis of the valve body portion, and means sealing the tie to the 70 movable wall.

The bore of the guide means and the crosssection of the valve body part may be of polygonal shape but preferably they are of substantially elliptical shape, and the major axis of the ellipse preferably extends in a substantially vertical plane in use to provide maximum support against tilting of the movable wall.

Usually the valve body part of known booster assemblies of the kind set forth was of hollow cylindrical shape and the aperture in the housing 80 was defined by a rearwardly extending tubular extension of the housing, a combined annular seal and guide assembly being located radially between the radially inner surface of the tubular housing extension and the radially outer surface of the valve body. A rubber boot was then secured between the outer surface of the tubular housing extension and the rear end of the valve body to assist in preventing dust from reaching the seal and guide assembly. It was necessary, however, for this boot to be perforated to accommodate the change in volume of the air trapped within the boot which occurs in use. Thus some dirt inevitably reached the seal and guide assembly 95 and sometimes led to failure of the seal with consequent loss of differential pressure in the housing.

According to a second aspect of the invention in a servo booster assembly of the kind set forth the guide means comprises an integral part of the housing that engages directly with the outer surface of the valve body part, and the valve body part is sealed to the housing by a rolling diaphragm located rearwardly of the guide means and arranged to roll from an external surface of the valve body part to an internal surface of the housing on forward movement of the valve body relative to the housing.

The rolling diaphragm, in addition to effecting a 110 seal of the rear chamber of the housing will also prevent dirt from contaminating the engaging surfaces.

With conventional boosters where the seal is a sliding seal with the outer surface of the valve 115 body, it has been necessary to ensure that the valve body is provided with a high quality outer surface. When glass fibre-reinforced plastics are used there exists a great danger that glass fibres protruding through the valve body outer surface 120 may damage the seal. When a plastics valve body has been used this has necessitated the use of expensive thermosetting plastics. The invention enables the use of thermoplastics materials for the valve body leading to a saving in weight. A 125 moulded thermoplastics valve body will have a rougher surface than that of a thermosetting plastics one, but this is an advantage for guiding purposes since grease is better retained on the

valve body surface engaged by the housing.

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Preferably the surface of the housing that engages with the valve body is provided with circumferentially spaced axially extending grooves to reduce friction forces resisting movement of the valve body.

Preferably the housing is formed of moulded plastics and said grooves are produced during moulding of the housing.

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Preferably the valve body projects rearwardly of the housing in the retracted position of the valve body, and the rolling diaphragm is provided with a first peripheral bead which is secured over the rear end of the housing and with a second peripheral bead which is held within the rear end of the valve body by a retaining ring of C-shape in cross-section, the free end of one limb of the retaining ring having snap engagement in an internal annular recess spaced axially from the rear end of the valve body.

When the outer surface of the valve body part is non-round in cross-section in accordance with the first aspect of the invention and a rolling diaphragm is employed in accordance with the second aspect of the invention, an elliptical outer surface of the valve body part is preferable to a polygonal outer surface, since a rolling diaphragm is not suitable for use with polygonal surfaces.

The invention will now be further described by way of example only with reference to the accompanying drawing in which:--

Figure 1 is a longitudinal cross-section of a vacuum servo booster in accordance with the invention;

Figure 2 is an enlarged view of the rear end of the booster of Figure 1;

Figure 3 is a cross-section on the line 2-2 of Figure 1, but omitting details of the poppet valve;

Figure 4 is a view similar to Figure 3 but of a modification. 40

With reference to Figure 1, the booster comprises a housing formed of moulded plastics shells 1 and 2 between the mating edges of which is held a peripheral bead 3 of a rolling diaphragm 4 secured to a metal diaphragm support plate 5. The plate 5 is provided with a rearwardly extending tubular portion 6 rigidly secured over the head 7 of a tubular valve body 8 moulded from a thermoplastics material. A pair of diametrically opposed metal tie rods 7 extend axially through both shells 1 and 2 and through the diaphragm plate 5 which is sealed to the rods 7 by resilient sliding seals 9. The tie rods 7 transmit in use brake reaction forces from a master cylinder housing to which they are secured at their front ends to a vehicle bulkhead to which they are secured at their rear ends.

The housing shell 2 is provided with a rearwardly extending integral tubular extension 10 having a front portion 11 that engages directly with the outer surface 12 of the rear part 13 of the valve body to guide the valve body for axial sliding movement with the diaphragm support plate 5, and a rear portion 14 spaced radially from surface 12. The portion 11 constitutes a guide

means for the valve body.

With reference to Figure 2, the rear end of the portion 14 of the housing extension 10 is provided with a radially outwardly extending flange 15 which retains a first peripheral bead 16 of a rolling diaphragm 17, the bead 16 being under sufficient circumferential tension to form an effective seal with the housing extension 10. A second peripheral bead 18 is located within the 75 rear end of the valve body 8 by a retaining member 19 which effectively comprises an annular ring 20 of C-shape in transverse crosssection. The ring portion 20 has limbs 22, 23 and 24, which are dimensioned such that the free end of limb 22 is a snap fit in an internal annular recess 25 is part 13 and such that the second bead 18 is held radially outwardly against part 13 with the adjacent portion 26 of the diaphragm held forwardly against the free end of part 25 to effect efficient seals therewith.

The intermediate portion 27 of the diaphragm rolls in use from the outer surface 12 of the valve body 8 to the elliptical inner surface of the rear portion 14 of the housing on forward movement 90 of the valve body 8 relative to housing shell 2. The inner surface or rear part 14 may be of any suitable shape provided that it does not possess any discontinuity in its curvature.

The end of the limb 22 and that of the valve 95 body are provided with chamfers 28 and 29 respectively to assist the assembly of the retaining member 19 to the valve body.

Referring again to Figure 1, a pedal-operated input rod 30 engages at its front end with a valve 100 control member 31 provided at its rear with a first annular valve seat 32 co-axial with a second annular valve seat 33 formed on the valve body. the valve seats 32 and 33 being engageable by an annular poppet valve member 34 in the usual 105 manner.

As shown in Figure 3, the outer surface 12 of the part 13 of the valve body is of elliptical crosssection throughout its length, and the internal surface of the portion 11 of the housing extension 110 10 is of complementary shape except that the portion 11 is formed with a series of circumferentially spaced axially extending recesses 35 to reduce the area of contact between portion 11 and surface 12 and to vent 115 the space 14' defined between rear portion 14, surface 12 and rolling diaphragm 17. The radially inner surface 36 of valve body portion 13 is of circular cross-section, and the major axis of the elliptical outer surface is vertical such that the 120 thickened prtions of the body part 13 are disposed at the top and bottom to provide maximum support against tilting of the diaphragm plate under its own weight and under the vertical force components exerted by the pedal linkage. 125 The vacuum conduit channels 37 are also conveniently located in the thickened portions of

body portion 13. In the modification of Figure 4 the portion 11 of housing extension 10 has an elliptical outer 130 surface rather than the round one of Figure 3.

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### Claims.

1. A servo booster assembly of the kind set forth in which the gide means has a bore of non-round cross-section, and the outer surface of the valve body part is of non-round cross-section in a plane transverse to the direction of movement of the valve body relative to the housing so as to resist turning of the valve body and movable wall relative to the booster housing.

2. A servo booster assembly as claimed in claim 1 in which a tie extends through the movable wall from a wall of the housing on one side of the movable wall to a further wall of the housing on the opposite side of the movable wall, the tie, at least where it passes through the movable wall, having its axis, displaced from the longitudinal axis of the valve body part, and means sealing the tie to the movable wall.

3. A servo booster assembly as claimed in
 claim 1 or claim 2 in which the bore of the guide means and the cross-section of the valve body part are of substantially elliptical shape.

4. A servo booster assembly as claimed in any of the preceding claims in which the inner surface
25 of guide means and/or the outer surface of the valve body part is provided with recesses to reduce the area of contact between the guide means and the valve body part.

5. A servo booster assembly as claimed in any of the preceding claims in which the guide means comprises an integral part of the housing that engages directly with the outer surface of the valve body part, and the valve body part is sealed to the housing by a rolling diaphragm located rearwardly of the guide means and arranged to roll from an external surface of the valve body part to an internal surface of the housing on forward movement of the valve body relative to the housing.

6. A servo booster assembly as claimed in claim 5 in which the valve body part projects rearwardly from the housing in the reatracted position of the valve body, and the rolling diaphragm is provided with a first peripheral bead which is secured over the rear end of the housing and with a second peripheral bead which is held within the rear end of the valve body by a retaining ring of substantially C-shape in cross-section, the free end of one limb of the retaining ring having a snap engagement in an internal recess spaced axially from the rear end of the valve body.

7. A vehicle having mounted therein a servo booster assembly in accordance with claim 3, or any of claims 4 to 6 each as appended to claim 3, in which the major axis of the elliptical cross-section of the valve body part is orientated substantially vertical.

 A servo booster assembly substantially as
 described with reference to Figures 1 to 3 of the accompanying drawings.

9. A servo booster assembly as claimed in claim 8 and modified substantially as described with reference to Figure 4 of the accompanyingdrawings.