

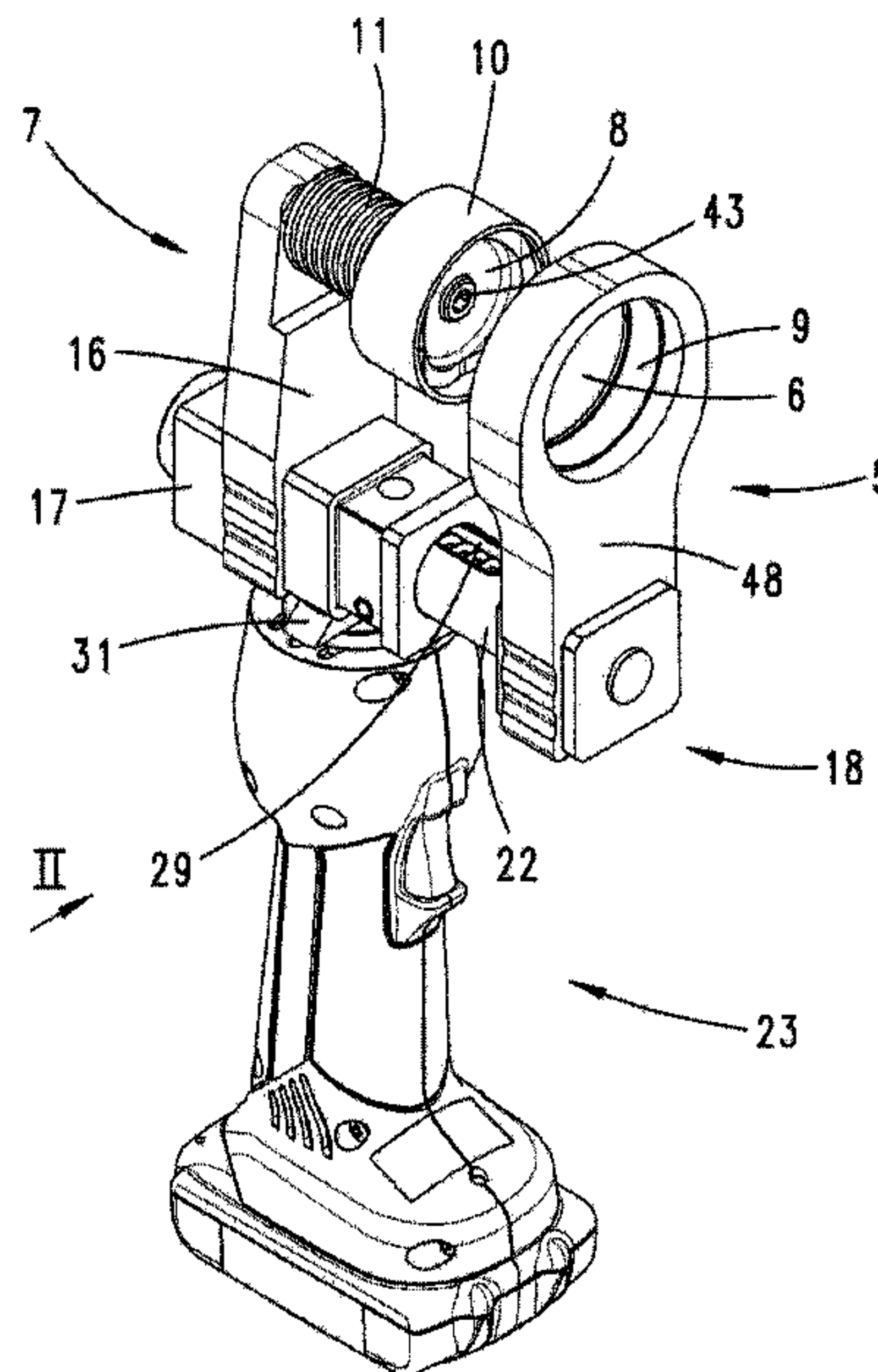


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(54) Titre : PAIRE DE MACHOIRES POUR L'ESTAMPAGE DE TROUS
 (54) Title: JAW PAIR FOR PUNCHING OUT HOLES

Fig. 1



(57) **Abrégé/Abstract:**

The invention relates to a jaw pair for punching out holes or openings, preferably in sheet metal parts, which jaw pair can be connected to a device (23) to be operated preferably hydraulically and which is designed for example as a pivoting jaw pair, wherein a punch (8) is formed on one jaw and a punching opening (6) is formed on the other jaw. In order to specify an advantageous jaw pair for punching out holes or openings, preferably in sheet metal parts, a scraper part (10) is formed in the jaw having the punch (8), which scraper part can be moved relative to the punch (8) and which is located outside of an outer contour of the punch (8) in the punching direction.



1 ABSTRACT

2 The invention relates to a jaw pair for punching out holes or openings, preferably in
3 sheet metal parts, which jaw pair can be connected to a device (23) to be operated preferably
4 hydraulically and which is designed for example as a pivoting jaw pair, wherein a punch (8) is
5 formed on one jaw and a punching opening (6) is formed on the other jaw. In order to specify an
6 advantageous jaw pair for punching out holes or openings, preferably in sheet metal parts, a
7 scraper part (10) is formed in the jaw having the punch (8), which scraper part can be moved
8 relative to the punch (8) and which is located outside of an outer contour of the punch (8) in the
9 punching direction.

1 **Jaw Pair for Punching Out Holes**

2 The invention relates to a jaw pair for punching out holes or recesses, preferably in sheet metal
3 parts, formed as a pivoting jaw pair, for example, the jaw pair being connectable to a device to
4 be operated preferably hydraulically, and a punch being formed on one jaw and a punching
5 opening being formed on the other jaw.

6 Such jaw pairs are already known in various respects. Reference is made to DE 199 34 288 C1
7 and US 6,427,515 B1, for example.

8 In addition, hydraulically operated devices on which such a jaw pair can be mounted are known
9 in various embodiments. Reference is made in this regard to WO 99/04165 A1 and
10 US 6,206,663 B1, DE 198 54 943 A1, WO 99/19947 A1 and US 6,276,186 B1, EP 1 084 798 A2
11 and US 6,718,870 B1, and WO 03/084719 A2 and US 7,412,868 B2.

12 With regard to the known punching device, a favorable operation is the goal. After the punch
13 has broken through, there is sometimes jamming of sheet metal or other parts from which
14 punching-out has taken place.

15 Against this background, it is an object of the invention to provide an advantageous jaw pair for
16 punching out holes or recesses, preferably from sheet metal parts.

17 According to a first inventive concept, one possible approach to this object is obtained with
18 subject matter in which a stripper part situated outside of an outer contour of the punch in the
19 punching direction, and movable in relation to the punch, is formed on the punching jaw having
20 the punch. With such a jaw pair, in which one or both jaws execute(s) a movement in relation to
21 the device in the course of performing a punching operation, this stripper part achieves the
22 result that jamming of the device in the sheet metal part, for example, or entrainment of the
23 sheet metal part on the jaw of the jaw pair having the punch after the end of the punching
24 operation, can be counteracted. A sequence of punching operations can then be performed
25 efficiently one after the other without hindrance.

1 Additional features of the invention are explained below and illustrated in the description of the
2 figures and in the accompanying drawings, often in their preferred association with the concept
3 already described above, but they may also be of importance in association with only one or
4 more individual features, which are described or illustrated here, or independently, or in another
5 overall concept.

6 It is thus preferable that the stripper part is biased into a position at least partially protruding
7 beyond the punch at its free end. This bias makes it possible for the stripper part to move
8 backward when the jaws come together, when the stripper part initially comes to rest on the
9 workpiece, until the punch acts on the workpiece. Also in the course of this action, the stripper
10 part initially moves farther backward, corresponding to the position of the punch in relation to the
11 workpiece into which it is penetrating and/or through which it penetrates. After the end of the
12 punching operation, when the jaws are moving apart again, the stripper part resting on the
13 (remaining) workpiece assists the movement of the punch away from the workpiece. Unless the
14 bias force is exceeded, the workpiece is reliably lifted off the punch.

15 The stripper part may fundamentally be designed, for example, in the form of a plurality of
16 stripper rods arranged preferably symmetrically and with equal circumferential spacing around
17 the punch. However, a sleeve-like design of the stripper part is preferred. In this design, any
18 remaining small parts are also reliably pushed back and ultimately stripped off when the jaws
19 are opened.

20 It is also preferred that the bias of the stripper part is achieved by a mechanical spring, in
21 particular a helical spring. The helical spring may be formed to surround a stationary securing
22 part and/or a mount of the punch. One possibility is a circular securing part, which is accordingly
23 surrounded by a helical spring extending in the manner of a cylindrical surface.

24 As an alternative to the formation of the jaw pair as a pivoting jaw pair, the jaw pair may also be
25 formed as a sliding jaw pair. It has advantageously been found that the punch and the punching
26 opening move parallel to one another, with coinciding central axes, in contrast to a movement
27 on the (same) arc in the case of a pivoting jaw pair.

1 In another configuration of a sliding jaw pair, it is preferably provided that the punch jaws are
2 accommodated in mutually displaceable mounts, and these mounts have a shared connecting
3 fitting to the device. In particular with respect to a hydraulically operated device, this connecting
4 fitting then also has the required hydraulic interface. The movement apparatus of the jaws of the
5 sliding jaw pair is preferably provided on the jaw side of the connecting fitting. In this
6 embodiment, the sliding jaw pair, which is exchangeably connectable to such a device, is
7 provided together with the movement apparatus, preferably hydraulic, for the displacement of
8 the jaws in relation to one another.

9 It is preferable in particular that the movement apparatus is a hydraulic piston/cylinder
10 mechanism.

11 It is further preferable that the cylinder is movable, but the piston is stationary. It is likewise
12 preferable that one jaw, more preferably the jaw having the punch, is stationary in relation to the
13 device during a punching operation in the case of the sliding jaw pair, whereas the other jaw,
14 preferably the jaw having the punching opening, is displaced in relation to the device. For an
15 elongated device, as is also preferred in this context, the displacement is preferably carried out
16 transverse to a longitudinal axis of the device, more preferably at right angles to the longitudinal
17 axis of the device.

18 It is preferable that the movable cylinder part has an end which can be acted upon by hydraulic
19 means and cooperates with the stationary piston part, and has a dry portion which extends
20 beyond a bottom area of the piston part in the open position of the jaws and is similarly carried
21 along when a punching stroke is performed, and which accordingly comes into partial overlap
22 with the piston when the punching stroke is performed. The return spring, which causes the
23 jaws to move back into their starting position after the hydraulic pressure is released, is
24 preferably accommodated in the dry portion.

25 It is also preferable that one of the jaws, i.e., the jaw having the punch or the jaw having the
26 punching opening, is fixedly connected to the stationary part or to the movable part of the
27 movement apparatus.

1 The ranges and value ranges given above and in particular below also include all the
2 intermediate values with regard to the disclosure, in particular in one-tenth increments of the
3 stated dimension, optionally also dimensionless, for example, 1/10 degree, 1/10 diameter, etc.,
4 on the one hand to delimit the specified range limits from above and/or from below, or
5 alternatively or additionally, also with regard to the disclosure of one or more individual values
6 from a respective stated range.

7 The invention is explained in greater detail below with reference to the appended drawings,
8 which, however, merely represent exemplary embodiments, in which

9 Figure 1 shows a perspective view of a device having a mounted sliding jaw pair;

10 Figure 2 shows a side view of the device according to Figure 1, in the upper
11 region;

12 Figure 3 shows a cross section of the subject matter according to Figure 2, in a
13 sectional view along line III-III;

14 Figure 4 shows a cross section of the subject matter according to Figure 2 and
15 Figure 3, in a sectional view in the plane IV-IV in Figure 3;

16 Figure 5 shows an illustration according to Figure 4 of the jaws, in the moved-
17 together state in the course of a punching operation;

18 Figure 6 shows a perspective view of the jaw pair obliquely from below, mounted
19 on the device according to Figure 1;

20 Figure 7 shows a perspective view of an alternative jaw pair formed as a pivoting
21 jaw pair;

22 Figure 8 shows a side view of the jaw pair according to Figure 7;

1 is preferably used here, also for the purpose of incorporating features described in the WO and
2 US documents with regard to the design of the device into claims of the present patent
3 application.

4 The jaw pair of the first embodiment is explained in further detail with respect to Figures 2 to 6.

5 A first jaw 5 has a punching opening 6 and the second jaw 7 has the punch 8.

6 In further detail, as shown in Figure 4, for example, the punching opening 6 is formed by a
7 punch insert 9, which may be a metal ring part hardened by a suitable method, for example.

8 The punch 8 in the exemplary embodiment is formed so that it is circular in plan view, so that
9 the punching opening 6 is also accordingly circular in plan view.

10 An angular design is also possible, such as that known from DE 10 2010 036 482 A1 (cf.
11 Figure 1 therein).

12 The punch 8 is surrounded by a stripper part 10, which is movable in relation to the punch 8
13 against the action of the compression spring 11 in the exemplary embodiment.

14 As shown in particular by a comparison of Figures 4 and 5, a front bordering edge 12 of the
15 stripper part 10 is preferably formed overall as a sleeve in the exemplary embodiment and is
16 arranged so that it is flush with one or more of the punch tips 13 formed on the punch 8 or also
17 protrudes beyond them. The punch tips 13 are preferably formed on the edge, based on a
18 contour outline of the punch 8. However, the stripper part 10 with its front bordering edge in the
19 starting position may also be arranged so that it is set back in relation to a front region of the
20 punch 8, in particular with respect to a punch tip 13. The punch may be formed with a profile in
21 its front surface, as described, while the stripper part preferably extends with its front edge in a
22 plane extending perpendicular to the punching direction.

23 In the exemplary embodiment, the stripper part 10 also has a bottom portion 14 with which it
24 engages behind the punch 8. In this regard, a form-fitting mount is also achieved to prevent
25 pulling off in the forward direction beyond the punch 8.

1 The punch 8 is also secured to a mount part 15 which extends in the punching direction and
2 forms a support. This mount part 15 is preferably cylindrical in the exemplary embodiment. The
3 compression spring 11 surrounds the mount part 15, preferably configured as a helical spring.

4 The mount part 15 is in turn secured at its foot at its end facing away from the punch 8, on a
5 securing part 16 extending at an angle to the mount part 15. In the exemplary embodiment and
6 preferably, the securing part 16 extends at right angles to a longitudinal axis of the mount part
7 15.

8 By means of the securing part 16, the punch 8 is ultimately secured on the sliding device 18,
9 preferably on a housing 17 of the sliding device 18 of the jaw pair.

10 The jaw having the punching opening 6 merges directly into a mount 48, in a manner of
11 speaking.

12 In further detail, the sliding device 18 comprises a piston part 19 and a cylinder part 20, and
13 further comprises a return spring 21.

14 The piston part 19 and the cylinder part 20 are accommodated concentrically with respect to
15 one another in the housing 17. A receptacle 22 in which the return spring 21 is accommodated
16 is formed in an extension of the cylinder part 20. The cylinder part 20 and the receptacle 22 are
17 formed in one piece or are connected for movement together.

18 Whereas the piston part 19 is stationary in relation to the device 23 when operated, as is also
19 the case for the housing 17, the cylinder part 20 moves together with the receptacle 22. To this
20 end, hydraulic fluid is pumped out of the device 23 and into the piston part 19 through a
21 connecting line 24, forming a hydraulic connection, for which purpose in particular a bottom
22 borehole 25 of the piston part 19 is connected to the connecting line 24.

23 In further detail, the piston part 19 is designed in the form of an elongated cylindrical part
24 having, in its interior, a recess 26, which is also cylindrical, as can be seen here. The elongated
25 design of the piston part 19, wherein the length L of the piston part preferably corresponds to

1 approximately two to five times the diameter D of the piston part, ensures stable guidance of the
2 cylinder part 20 which is movable in relation thereto.

3 Furthermore, it is apparent that the housing 17 has a cross bolt 27 which at the same time also
4 forms a support for the return spring 21 at one end. At the other end, the return spring 21 is
5 supported on a base 28 of the receptacle 22. With respect to the cross bolt 27 or a location of
6 the housing 17 in this regard, the piston part 19 and the return spring 21 extend in opposite
7 directions.

8 The jaws are secured directly on the housing 17 by means of the securing part 16 and the
9 mount 48, and are thus stationary with respect to the device 23 in the connected state; i.e., the
10 jaws are secured on the receptacle 22 by means of the mount 48. The jaws are fixedly secured.
11 The securing part 16 and the mount part 48 cannot move in relation to the housing 17 and the
12 receptacle 22, respectively. The described mounts of the jaws, which are connected to one
13 another by means of the sliding device 18, together with the sliding device 18, are also
14 connected or can be connected to the device 23 by means of a shared connecting fitting 30.
15 The piston/cylinder unit of the sliding device 18 is formed on the jaw side of the connecting
16 fitting 30.

17 As a result of pressurized filling of the recess 26 with hydraulic fluid, the cylinder part 20 is
18 moved out of the position according to Figure 4 into the position according to Figure 5. Since the
19 receptacle 22 is also moved in this way, the return spring 21 is correspondingly compressed
20 and biased.

21 The cross bolt 27 is arranged in a portion of the housing 17 which extends beyond the piston
22 bottom with respect to the connecting line 24, and engages through two oppositely situated
23 elongated holes 29 in the receptacle 22.

24 The connecting line 24 is part of the connecting fitting 30 (also see Figure 6), which permits an
25 operationally detachable connection of the unit of the jaws and the sliding device from the
26 device 23. The connecting fitting 30 can be screw-connected to the device.

1 In particular, the connecting fitting accordingly comprises the mechanical adapter, the screw
2 connection, for example, and a cavity which merges into the connecting line 24 for conducting
3 hydraulic fluid into and out of the piston part 19. The connecting fitting 30 at the same time
4 provides, via a neck-shaped configuration, that the mentioned unit of the jaws and the sliding
5 device is at a distance from a facing end face of the device 23. However, as in the exemplary
6 embodiment, the connecting fitting is preferably screwed into this end face of the device 23 with
7 the foot region having the threaded mechanism.

8 In addition, the jaw mechanism which is connected to the device 23 is supported overall on the
9 device 23 by means of an attachment part 31 having an elongated shape. The attachment part
10 31 extends transversely with respect to a displacement direction of the jaws.

11 The further embodiment having pivoting jaws is explained with reference to Figures 7 to 13.

12 The pivoting jaws 32, 33 are connected in a pivot joint 34. Beginning with the pivot joint, the
13 pivoting jaws 32, 33 on the device side are fundamentally configured in the same way as
14 described in DE 10 2005 028 083 A1 and US 7,216,532 B2. In this regard, the disclosure
15 content of this publication is also hereby incorporated in full in the disclosure of the present
16 patent application, also for including features of this previously known document [in claims] of
17 the present patent application.

18 The bearing eyes 35, 36 of each of the jaws 32, 33 are configured with different thicknesses as
19 measured in the axial direction of the bearing opening, as also described in the cited document.
20 The bearing eye 35 is approximately three times thicker than the bearing eye 36.

21 The inside distance between the two bearing eyes 36 and 35 corresponds approximately to the
22 thickness of the bearing eye 35 having the greater thickness. In the mounted state, the jaws are
23 aligned in relation to one another in such a way that the bearing eyes 35, 36 intermesh. The
24 bearing eye 35, having the greater thickness, of the one jaw 33 is correspondingly arranged
25 between the two bearing eyes 35 and 36 of the other jaw 32. Thus, the bearing eyes 36 having
26 the lesser thickness are situated on the outside of the jaw pair thus formed.

1 The mounting bolt 37 passes through the bearing openings 6 of the two jaws 32, 33 in the
2 mounted state.

3 As indicated in further detail from Figure 9, for example, the locking bolt 37 passes through two
4 arms 38, 39 of the mounting neck 40. The locking bolt 37 may in particular be a bolt such as
5 that known from EP 1 491 295 A1 and US 2005/0011236 A1. In this regard, the content of this
6 document is also hereby incorporated in full in the disclosure of the present patent application,
7 also for the purpose of including features of this document in claims of the present patent
8 application.

9 Toward the free end, i.e., toward the punch 8, the jaw 32 is formed differently from the jaw 33.
10 The jaw 32 has an angled shape in a side view according to Figure 8, such that a first jaw
11 portion extends approximately horizontally in the open state of the jaws according to Figure 7,
12 starting from the pivot joint 34, while a second jaw portion extends approximately vertically. The
13 punch 8 is then held on the other jaw portion, in principle in the same way as already described
14 with respect to the jaw of the first embodiment.

15 In contrast, the second jaw 33 already extends approximately vertically in the position according
16 to Figure 7 and Figure 8, the second jaw portion of the jaw 32 and the jaw 33 diverging from one
17 another, forming an angle α of approximately 30° to 60° .

18 The pivoting jaws of the second embodiment are explained in greater detail with reference to
19 Figures 10 to 13.

20 Figures 10 and 11 illustrate a connecting fitting of this embodiment. In this regard, it is important
21 that, corresponding to the teaching of EP 1 084 798 A2 and US 6,718,870 B1, a piston shank
22 50 surrounded by a return spring 51 is already situated in the connecting fitting 49 with respect
23 to the piston/cylinder arrangement formed in conjunction with the device 23. In this embodiment
24 as well, rollers 52 which cooperate with the corresponding acting surfaces 53 of the jaws are
25 accommodated in the connecting fitting 49 on the jaw end of the piston shank 50. In this
26 embodiment, the jaws are further preferably biased into a position, which basically corresponds
27 to the punching position, by a compression spring 54.

1 Figure 11 shows the position at the end of a punching operation, the piston 51 being
2 hydraulically displaced by the connected device 23, not shown in greater detail here.

3 Figures 12 and 13 show in perspective and cross sectional views, respectively, the arrangement
4 of the jaws without the connecting fitting and in their open position.

5 With respect to Figures 14 to 16, the design of the punch 8 and its mounting are also explained
6 in further detail with regard to two possible embodiments.

7 The embodiment shown in Figure 4, for example, is depicted in an exploded illustration in
8 Figure 14. The mount part 15 has a plate formation 41 on the punch end, protruding beyond the
9 cylindrical mount part 15 transversely with respect to a longitudinal extent of the mount part 15.
10 The largest possible surface for support of the mount part 15 on the facing rear side of the
11 punch 8 is provided. An upper stop for compression spring 11 may also be formed in this way.

12 This plate formation has a central screw opening 42, in which a securing screw 43 engages, the
13 securing screw passing through the punch 8 and thus securing it on the mount part 15.

14 The stripper part 10 is mounted on the punch 8 on the one hand by means of an elongated hole
15 44 and a screw 45, which engages therein, and on the other hand by the undercut bottom
16 portion 14. This is advantageous during assembly, for example, when the mount part 15 is not
17 yet connected to the securing part 16 and therefore the compression spring 11 does not yet
18 have any countersupport.

19 The further embodiment according to Figures 15 and 16 differs from the embodiment described
20 above, in that at the same time a screw-shaped protrusion 46 is formed on the mount part 15
21 facing the punch 8. The punch 8 has a threaded opening 47, which fits the former, so that the
22 two parts can be screw-connected.

23 All features disclosed are (in themselves) pertinent to the invention. The disclosure content of
24 the associated/accompanying priority documents (copy of the prior application) is also hereby
25 included in full in the disclosure of the application, including for the purpose of incorporating
26 features of these documents in claims of the present application. The subsidiary claims in their

- 1 optional subordinated formulation characterize independent inventive refinement of the prior art,
- 2 in particular to undertake divisional applications based on these claims.
- 3

1 List of reference numerals

2	1	Pressing device
3	2	Return valve
4	3	Tank
5	4	Pump ram
6	5	First jaw
7	6	Punching opening
8	7	Second jaw
9	8	Punch
10	9	Punch insert
11	10	Stripper part
12	11	Compression spring
13	12	Bordering edge
14	13	Punch tips
15	14	Bottom portion
16	15	Mount part
17	16	Securing part
18	17	Housing
19	18	Sliding device
20	19	Piston part
21	20	Cylinder part
22	21	Return spring
23	22	Receptacle
24	23	Device
25	24	Connecting line
26	25	Bottom borehole
27	26	Recess
28	27	Cross bolt
29	28	Base
30	29	Elongated holes
31	30	Connecting fitting
32	31	Attachment part

1	32	Pivoting jaw
2	33	Pivoting jaw
3	34	Pivot joint
4	35	Bearing eye
5	36	Bearing eye
6	37	Mounting bolt
7	38	Arm
8	39	Arm
9	40	Mounting neck
10	41	Plate formation
11	42	Screw opening
12	43	Securing screw
13	44	Elongated hole
14	45	Screw
15	46	Protrusion
16	47	Threaded opening
17	48	Mount
18	49	Connecting fitting
19	50	Piston part
20	51	Return spring
21	52	Roller
22	53	Acting surfaces
23	54	Compression spring
24	L	Length
25	D	Diameter
26	α	Angle

CLAIMS

1. A jaw pair for punching out holes or openings, preferably in sheet metal parts, which jaw pair is connectable to a device (23) to be operated preferably hydraulically and is designed, for example, as a pivoting jaw pair, wherein a punch (8) is formed on one jaw and a punching opening (6) is formed on the other jaw, characterized in that a stripper part (10) which is situated outside of an outer contour of the punch (8) in the punching direction is formed in the jaw having the punch (8) and is movable in relation to the punch (8).
2. The jaw pair according to claim 1 or in particular according thereto, characterized in that the stripper part (10) is biased into a position protruding at least partially beyond the punch (8) at its free end.
3. The jaw pair according to any one or more of the preceding claims or in particular according thereto, characterized in that the stripper part (10) has a sleeve-like design.
4. The jaw pair according to any one or more of the preceding claims or in particular according thereto, characterized in that the stripper part (10) is biased by a compression spring (11).
5. The jaw pair according to any one or more of the preceding claims or in particular according thereto, characterized in that the compression spring (11) surrounds a stationary mount part (15) of the punch (8).
6. The jaw pair according to any one or more of the preceding claims or in particular according thereto, characterized in that the jaw pair is formed as a sliding jaw pair.
7. The jaw pair according to any one or more of the preceding claims or in particular according thereto, characterized in that in the case of the sliding jaw pair, the jaws are accommodated in mutually displaceable mounts, and these mounts have a shared connecting fitting to the device.

8. The jaw pair according to any one or more of the preceding claims or in particular according thereto, characterized in that the connecting fitting (30) has a hydraulic connection.

9. The jaw pair according to any one or more of the preceding claims or in particular according thereto, characterized in that the mounts are displaceable in relation to one another by means of a hydraulic piston/cylinder unit formed at a jaw end of the connecting fitting.

Fig. 1

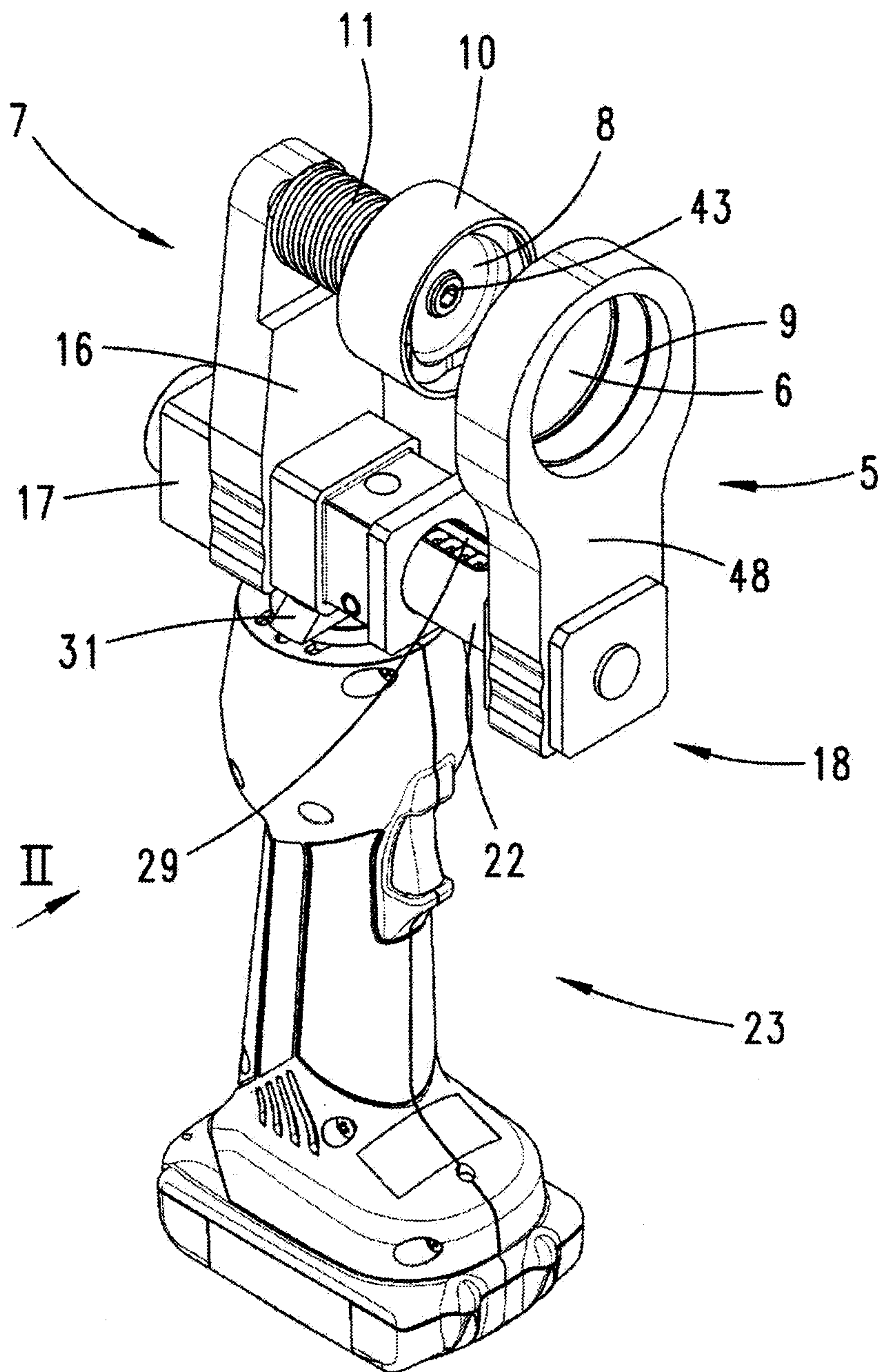


Fig. 3

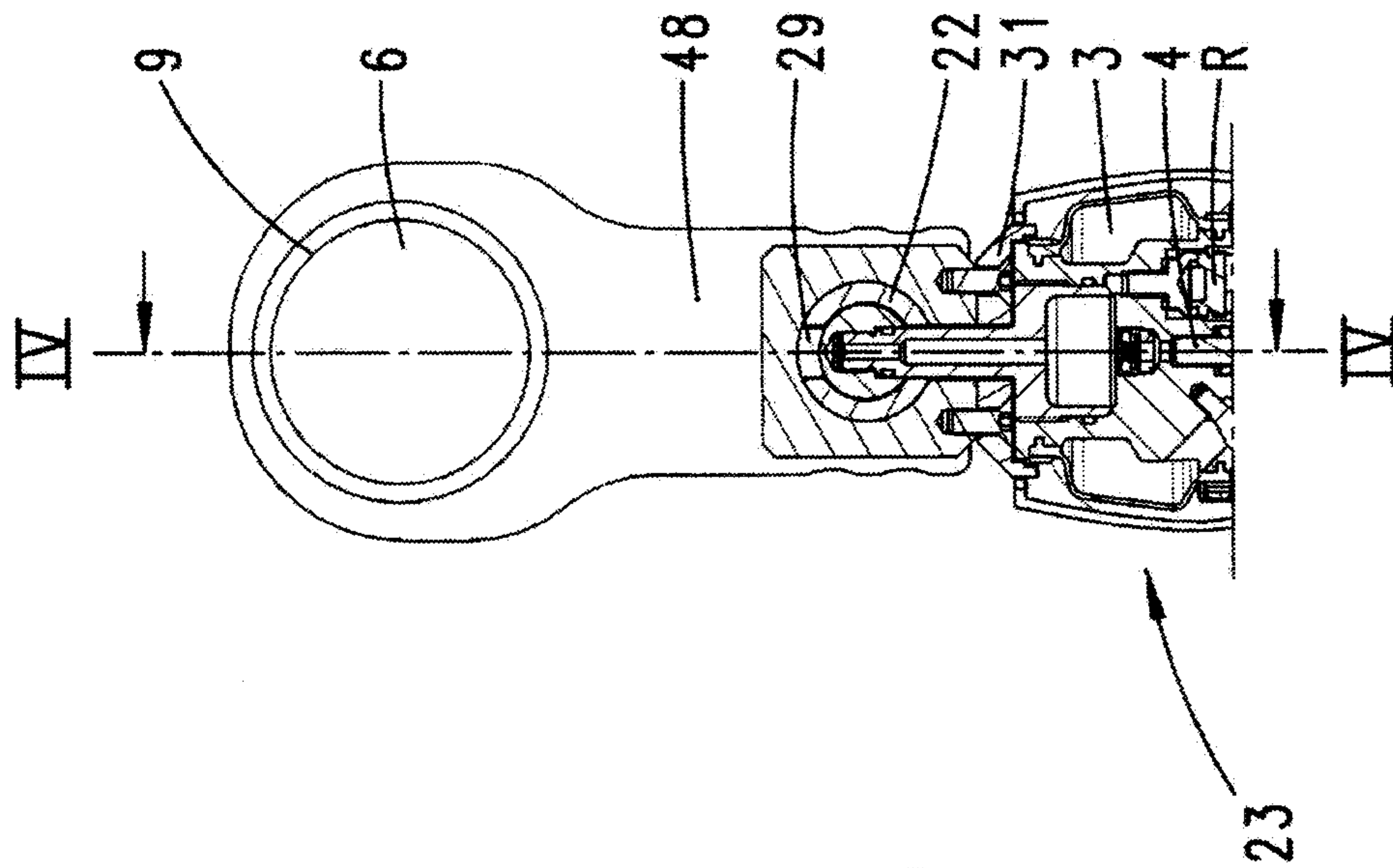


Fig. 2

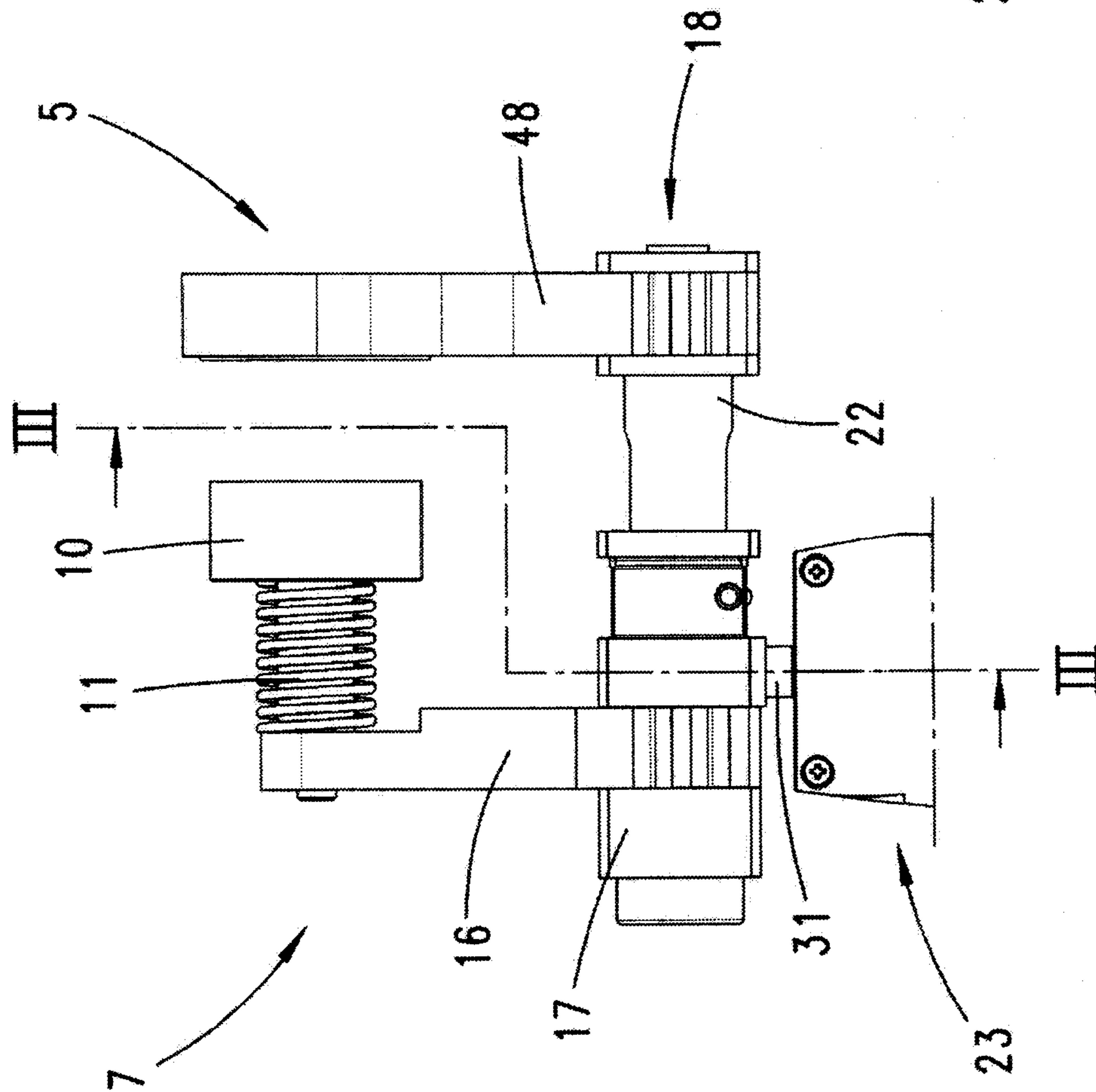


Fig. 5

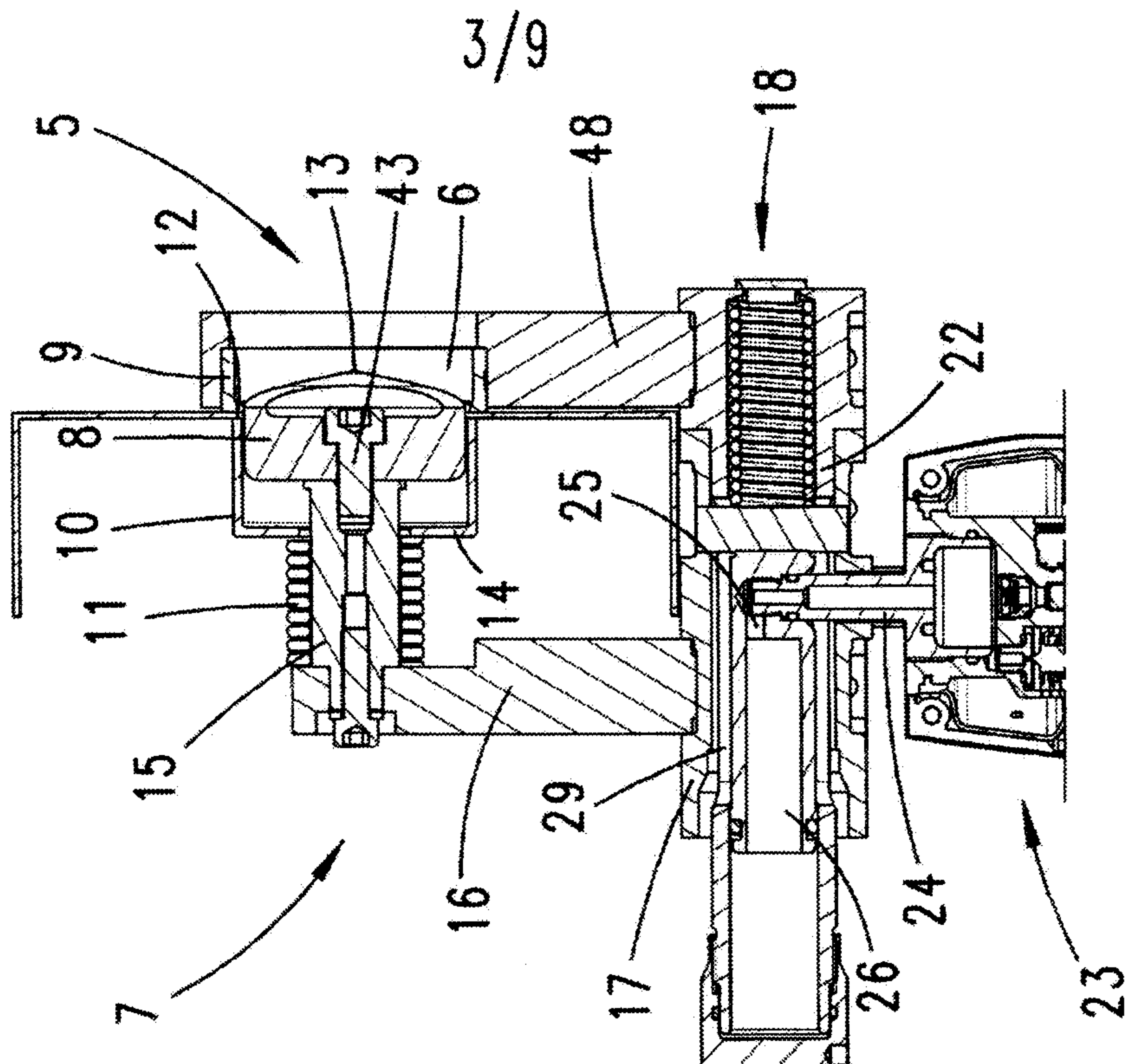


Fig. 4

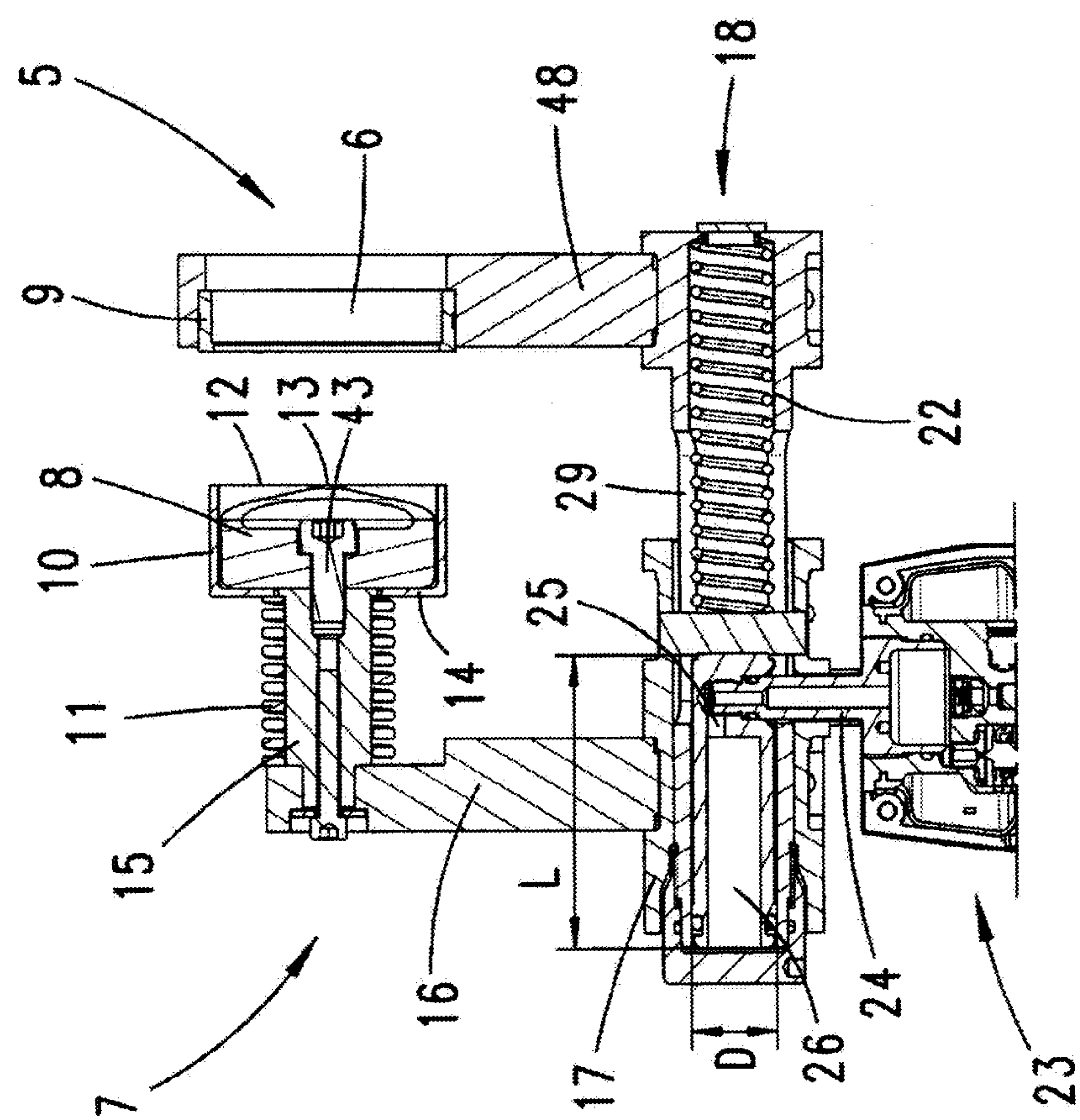


Fig. 6

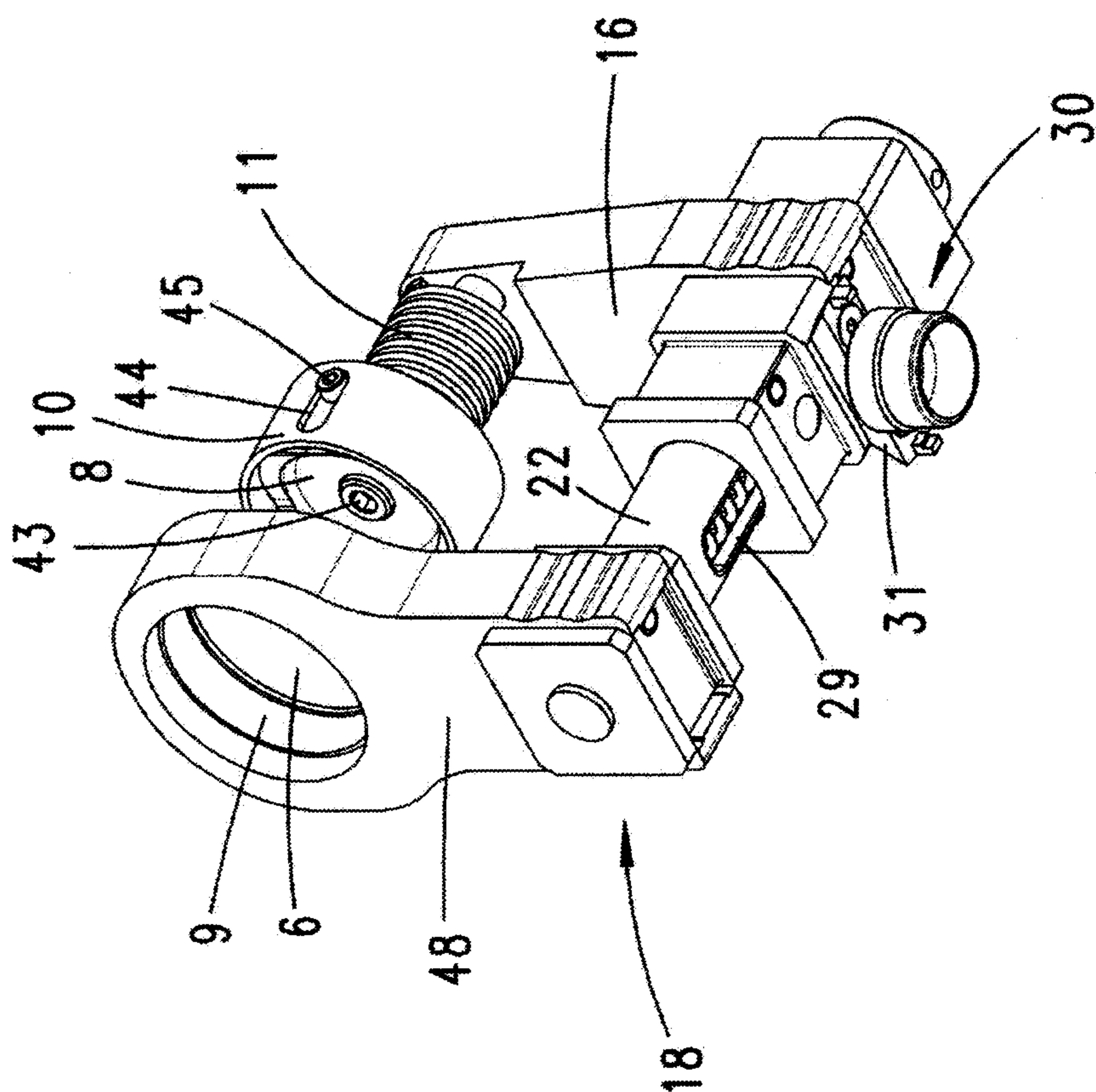
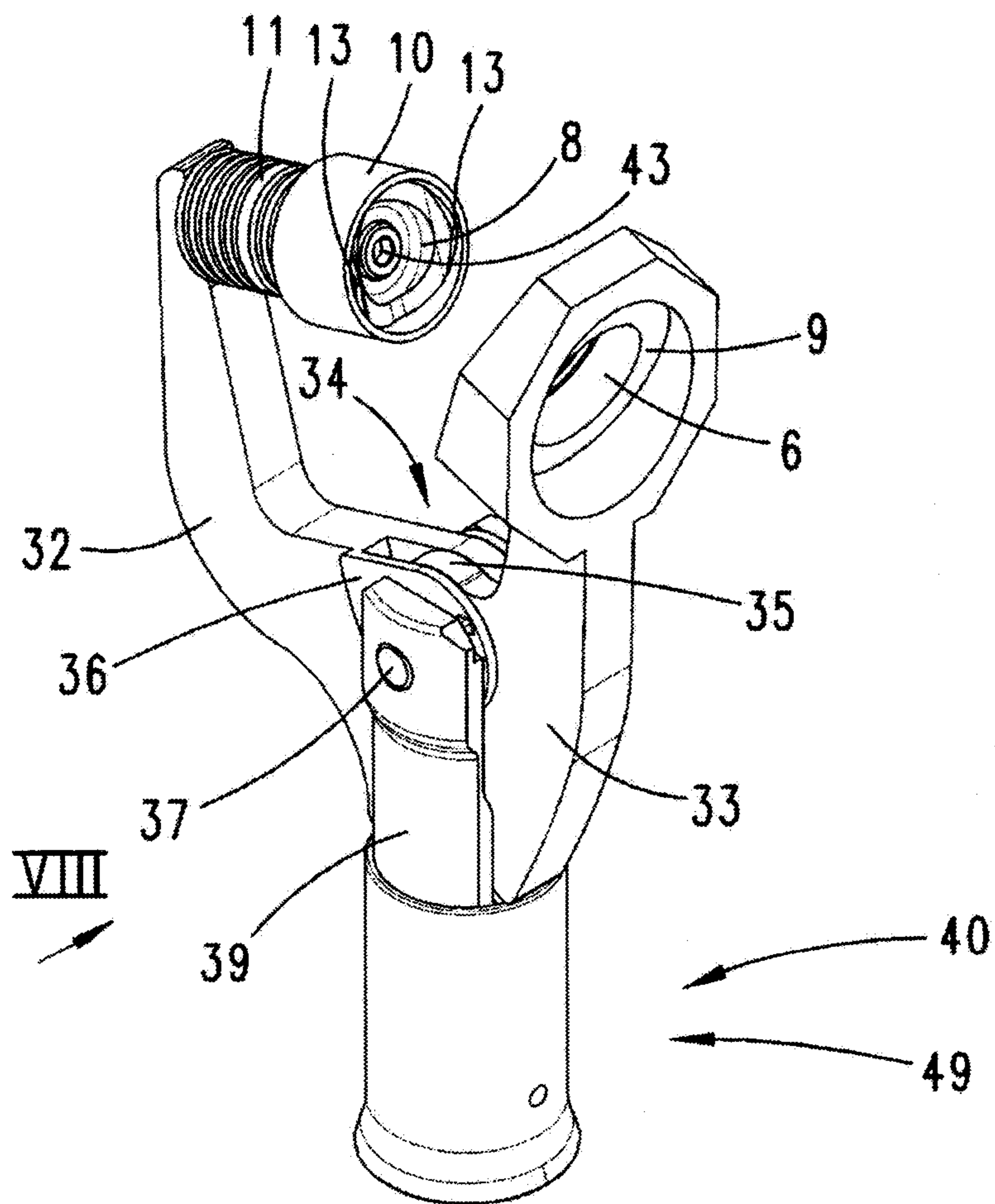
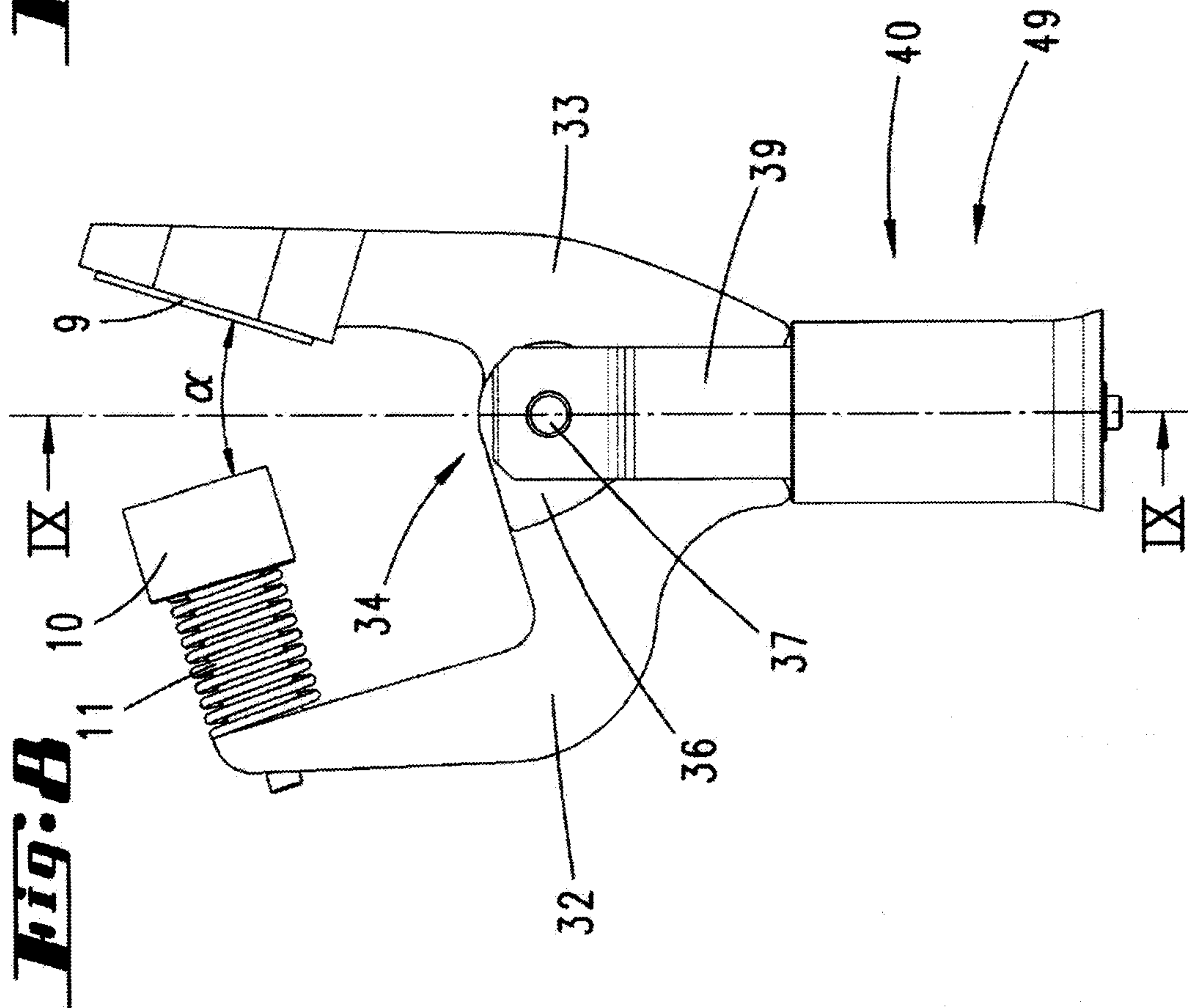
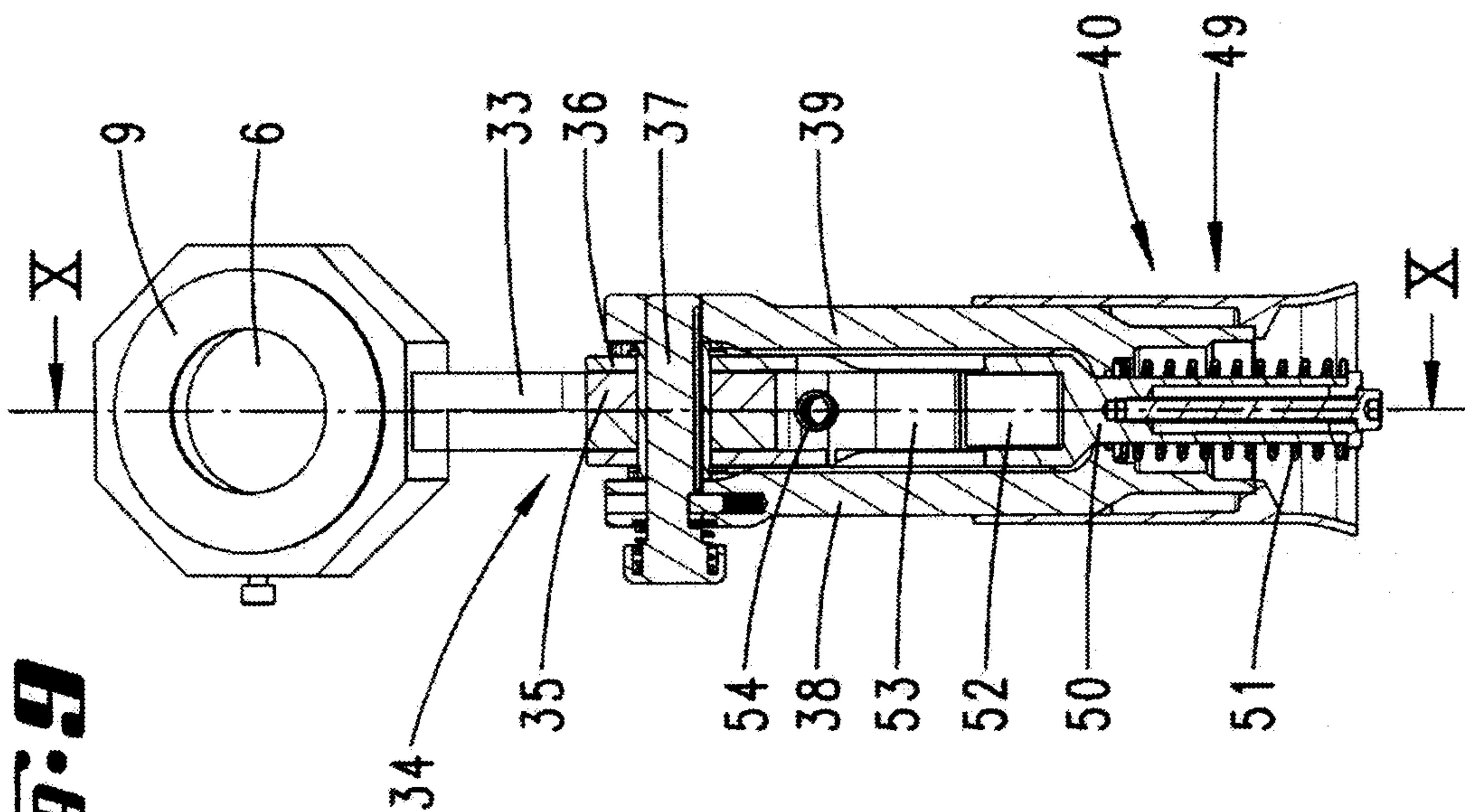


Fig. 7





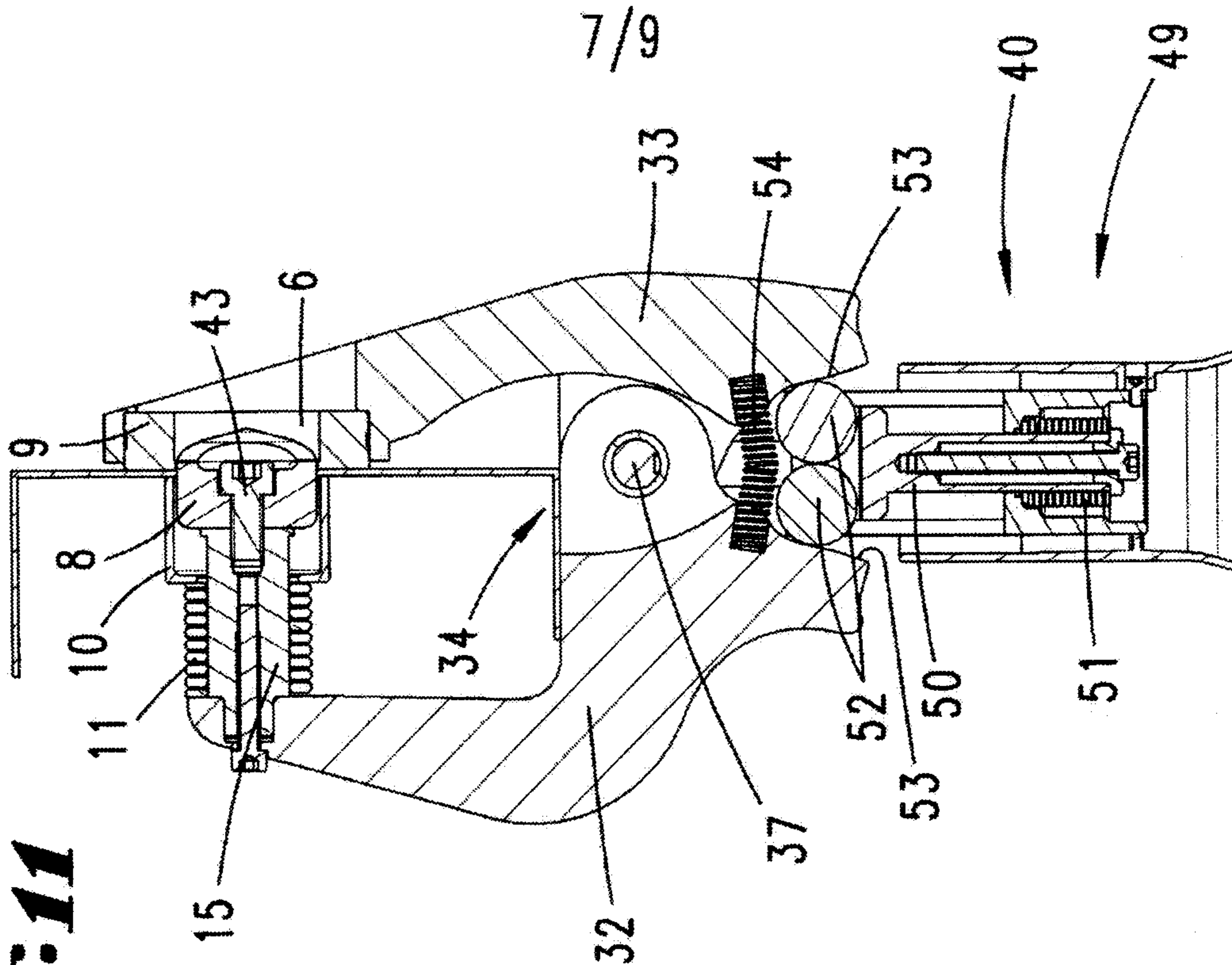


Fig. 11

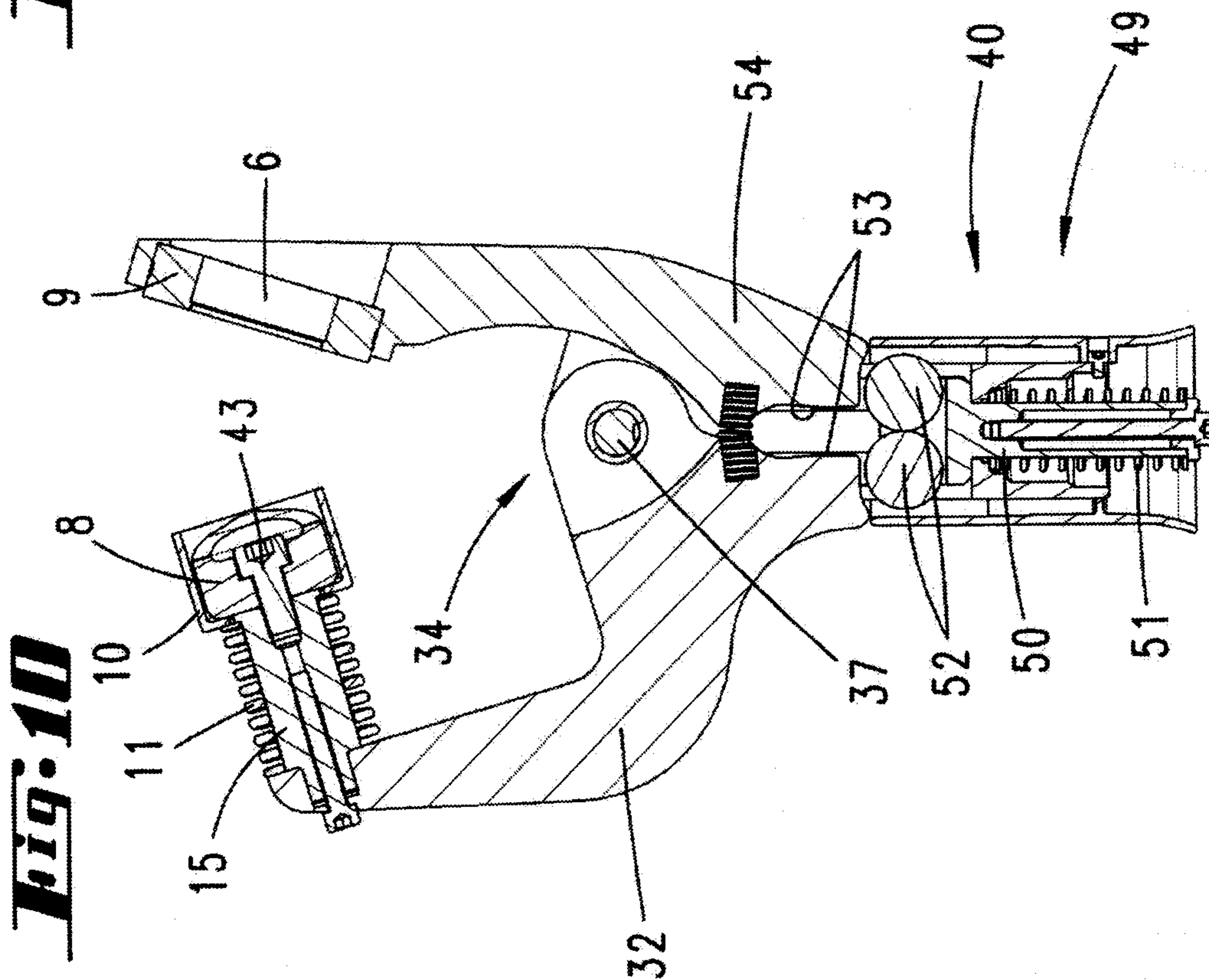


Fig. 10

Fig. 13

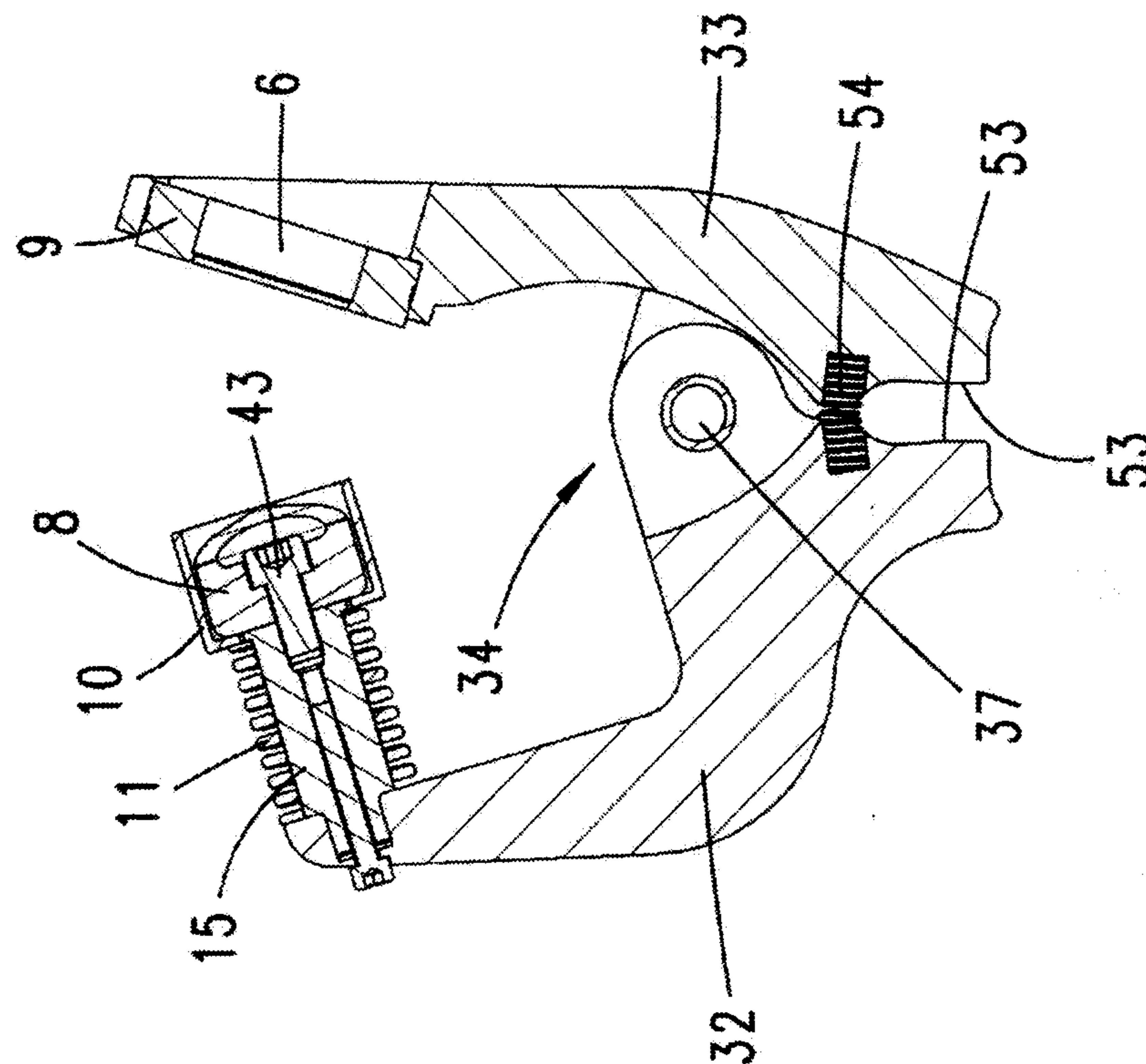


Fig. 12

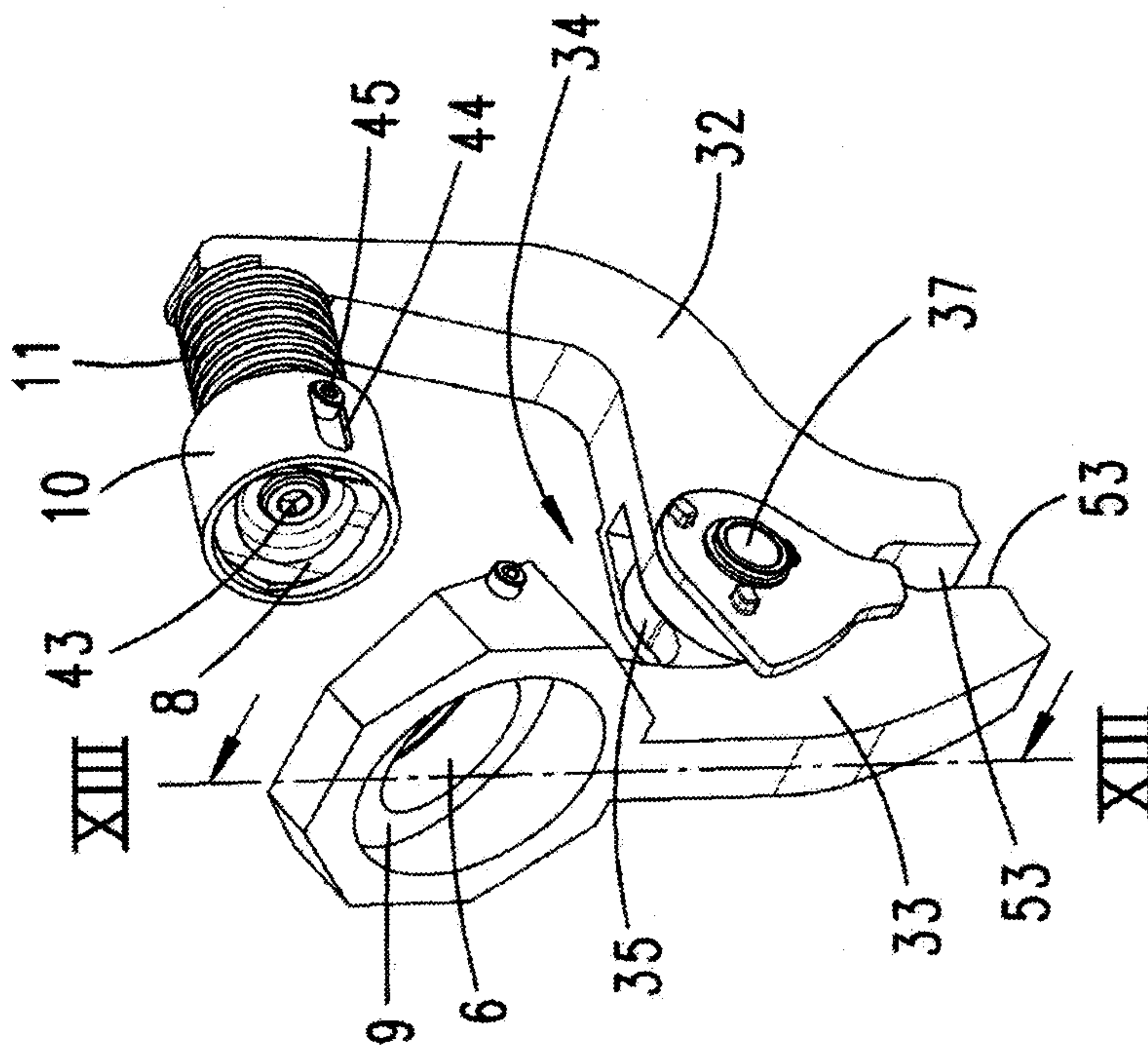


Fig. 16

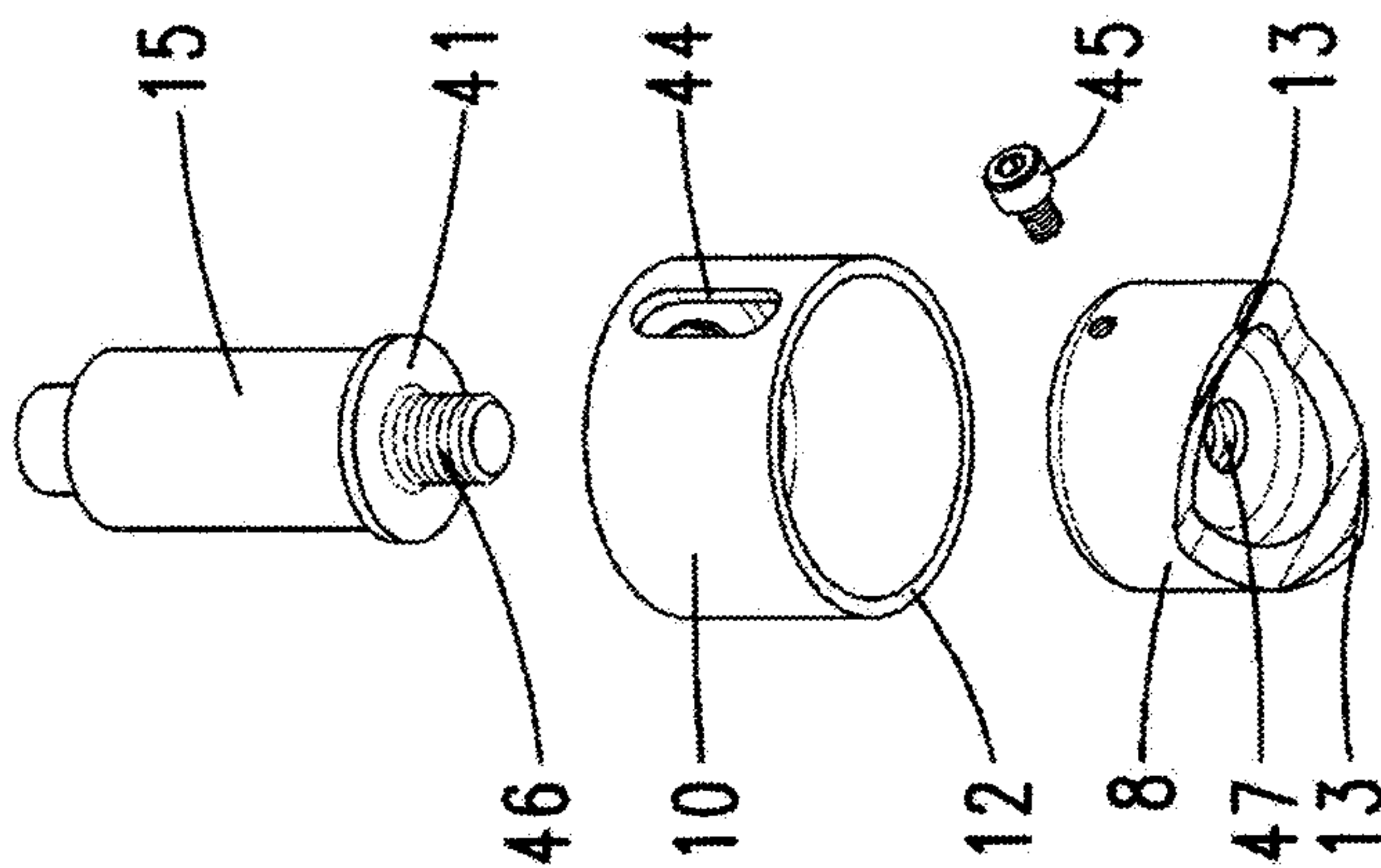


Fig. 15

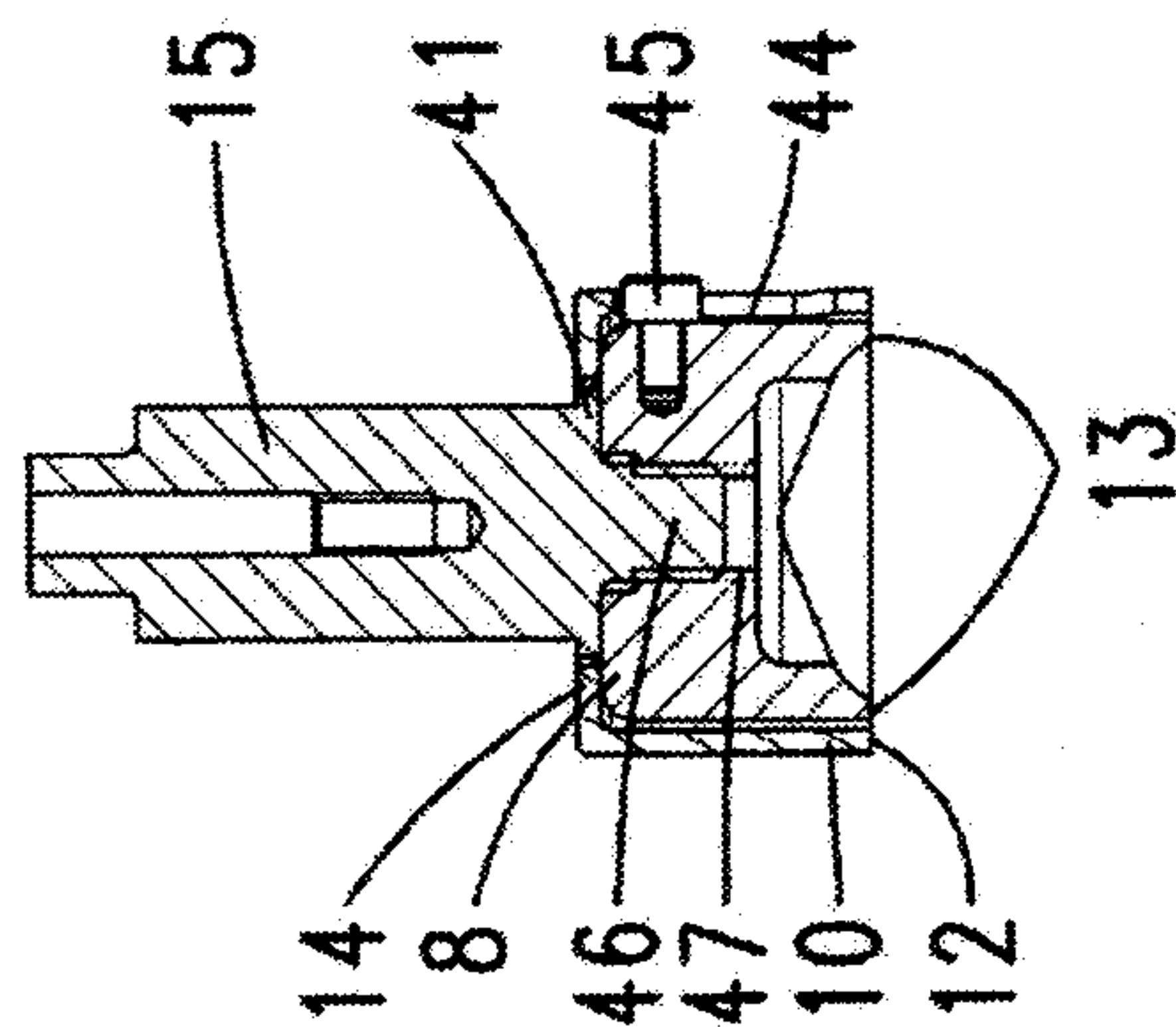


Fig. 14

