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(54) Title: MIXER HAVING ROTATING MIXING CONTAINER

(54) Bezeichnung : MISCHER MIT DREHBAREM MISCHBEHÄLTER

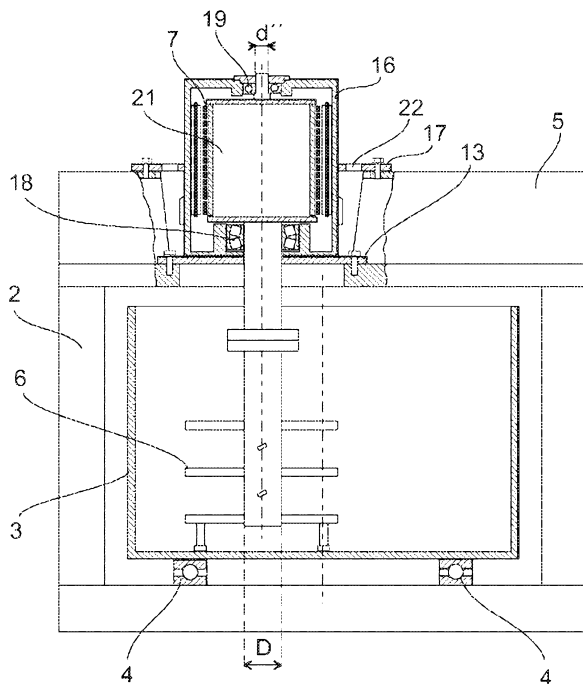


Fig. 2

(57) Abstract: The invention relates to a mixer having a rotating mixing container and a tool shaft (8) arranged at least partially in the mixing container, having a drive end and an operating end for a tool (6), wherein the tool shaft (8) is mounted on the drive end by means of two tool bearings, spaced apart from one another, and there is a drive motor (7) for the tool shaft (8). In order to create a mixer that is simple and cost-effective to produce, having as few movable and wearing parts as possible and a tool shaft (8) that is furthermore capable of sustaining the significant transverse forces occurring in operation, according to the invention the drive motor (7) has a motor shaft that is mounted at least on one side by a tool bearing.

(57) Zusammenfassung: Offenbart wird ein Mischer mit einem Mischbehälter und einer zumindest teilweise im Mischbehälter angeordneten Werkzeugwelle (8) mit einem Antriebsende und einem Arbeitsende für ein Arbeitswerkzeug (6), wobei die Werkzeugwelle (8) mittels zweier voneinander beabstandeter Werkzeuglager am Antriebsende gelagert ist und ein Antriebsmotor (7) für die Werkzeugwelle (8) vorgesehen ist. Um einen Mischer

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bereitzustellen, der einfach und kostengünstig herzustellen ist, der möglichst wenige bewegliche und verschleißende Teile aufweist und dessen Werkzeugwelle (8) zudem in der Lage ist, die im Betrieb auftretenden erheblichen Querkräfte aufzunehmen, wird vorgeschlagen, dass der Antriebsmotor (7) eine Motorwelle hat, welche zumindest einseitig von einem Werkzeuglager gelagert ist.

MIXER WITH ROTABLE MIXING CONTAINER

The present invention relates to a mixer with a mixing container and a tool shaft arranged at least partly in the mixing container, the tool shaft having a working end to which a working tool is fastened or can be fastened and a drive end which is mounted by means of two mutually set-apart tool bearings, and a drive motor being provided with a motor shaft for driving the tool shaft.

A mixer of this type is for example known from DE 35 20 409. The embodiment shown therein comprises a pressure-resistant mixer with a filling opening, a rotating mixing container having an emptying means, with mixing tools arranged eccentrically to the mixing container axis inside the mixing container.

The mixer known from the prior art is represented schematically in Figure 1 which is a vertical section through a mixer. The mixer 1 has a mixing container 3 which is received in a mixer housing 2 and can be rotated about a vertical axis of rotation. In order to ensure this rotation, the mixing container 3 is rotatably mounted on a ball bearing 4. The mixing container can have an emptying opening (not shown in the figure) at its underside. The mixer housing 2 has a housing cover 5. A working tool 6, which is embodied as a mixing tool, is arranged inside the mixing container 3. It may be seen that the working tool 6 is rotatable about a vertical axis which is set apart from the axis of rotation of the mixing container 3. For this purpose, a drive end of the working tool 6 is guided through the housing cover 5 and driven with the aid of the drive motor 7 via, for example, V-belts 9.

The working tools 6 are fastened to a tool shaft 8 which has a drive end on which the V-belt 9 acts and a working end to which the working tools 6, which are embodied as the mixing tool, are fastened. The tool shaft 8 is in the embodiment shown formed in two parts, wherein the two parts can be joined together or separated from each other via the flange connection 10. This flange connection 10 is provided inter alia to exchange the working tool 6 for another working tool 6, such as for example a star-type agitator for a pin-type agitator. In addition, the working tool can, when it displays phenomena of wear, be exchanged for a new one. As both the mixing container 3 and the tool shaft 8 rotate, considerable transverse forces can act on the tool shaft 8, the transverse forces being caused by the flow of material through the

rotating mixing container 3, especially as the tool shaft is held in the housing cover 5 only at one side. The magnitude of the transverse force depends inter alia on the nature of the material to be mixed and of course on the rotational speed of both the mixing container 3 and the working tool 6.

Two tool bearings 11, 12, which each mount the shaft having a diameter D , are therefore provided at the drive end for holding the tool shaft 8. In order to absorb the forces, the tool bearings 11, 12 are screwed to the mixer housing 2 or the housing cover 5 via a flange 13. The V-belt 9 then acts on the drive end of the tool shaft 8. The drive motor 7 has a motor shaft 20 which is also held via two motor bearings 14, 15. It may be seen that the diameter d' of the motor shaft 20 is much smaller than the diameter D of the tool shaft 8.

In the prior art, the drive motor is mainly in the form of three-phase asynchronous motors or hydraulic motors with V-belt or toothed belt transmission, and also geared motors.

All these types of drive have in common the fact that a large number of elements are required for generating torque and for converting torque and also for accommodating the load. In the simplest case of the asynchronous motor with a corresponding bearing arrangement, at least four bearings are required – two bearings for the motor shaft and two bearings for the tool shaft – which have to accommodate, as well as the weight forces, in addition also the high forces from the working tool and also the considerable belt forces.

If a geared motor or a separate gear mechanism is used, at least two further bearings have to be provided for each further reduction stage.

In addition to the complex and yet failure-prone bearings, the belt transmission, consisting generally of a set of several V-belts or toothed belts, is a high-maintenance machine element. These components have to be checked at regular intervals for correct stress and the stress must, if appropriate, be adapted. Likewise, both V-belts and toothed belts are prone to wear and must therefore be exchanged at regular intervals.

Object of the Invention

It is the object of the present invention to substantially overcome or ameliorate one or more of the above disadvantages.

Summary of the Invention

The present invention provides a mixer with a mixing container and a tool shaft arranged at least partly in the mixing container, the tool shaft having a working end to which a working tool is fastened or can be fastened and a drive end which is mounted by means of two mutually set-apart tool bearings, a drive motor being provided with a motor shaft for driving the tool shaft, wherein the motor shaft is mounted by at least one of the two mutually set-apart tool bearings.

In other words, one of the bearings, which is provided for mounting the tool shaft, is used at the same time for mounting the motor shaft. The motor shaft and tool shaft are therefore directly connected to each other. This measure allows at least one bearing to be avoided.

Particularly preferred is an embodiment in which the motor is arranged between the two tool bearings and the motor shaft is mounted preferably by means of the two tool bearings. This embodiment allows two bearings to be dispensed with, as the bearings for the tool shaft serve at the same time as bearings for the motor shaft. Basically, in this embodiment, it is no longer possible to distinguish between the motor shaft and tool shaft, as one portion of the shaft functions as the motor shaft and another portion of the same shaft functions as the tool shaft.

The motor used is in these cases preferably a direct drive and particularly preferably a three - phase synchronous motor (servo motor, torque motor, reluctance motor).

In a further preferred embodiment, the bearing of the motor shaft that faces the tool shaft is suitable for accommodating particularly high radial and axial forces. The bearing is designed preferably as a combined radial axial bearing (radiax bearing), for example as a self-aligning roller bearing or self-aligning ball bearing and particularly preferably a twin-row self-aligning roller bearing.

It has been found that, in particular, a twin-row self-aligning roller bearing can best accommodate the transverse forces occurring during operation.

A further preferred embodiment makes provision for the diameter of the motor shaft to differ on the two tool bearings, preferably the diameter of the motor shaft d'' on the tool bearing facing away from the tool shaft being smaller, preferably at least 30 %, particularly preferably at least 50 % smaller, than the diameter of the motor shaft D on the other tool bearing.

It has been found that merely the bearing facing the mixing container must have a large diameter. On suitable designing of the bearings, the bearing facing away from the mixing container can be made much smaller and thus more economical.

The motor is expediently arranged in a motor housing, both tool bearings being arranged on or in the motor housing. In this case, the motor housing can have a first outer flange by which

the motor housing, and thus the motor, is fastened to the mixer housing. Furthermore, in a particularly preferred embodiment, the motor housing can have a second outer flange which is also fastened to the mixer housing, the second outer flange having preferably a larger average diameter than the first outer flange.

5 The motor housing could have, for example, a circular cross-section, the outer flanges then expediently also having a circular cross-section. However, in principle, other cross-sections, for example square or rectangular cross-sections, are also conceivable. The fact that the second outer flange has a larger average diameter means that the motor can easily be fastened to the mixer housing. For example, the mixer housing can have a stepped through-opening
10 with a first portion having a smaller average diameter and a second portion having a larger average diameter, the second portion having an average diameter which is larger than the average diameter of the first outer flange and smaller than the average diameter of the second outer flange. In a preferred embodiment, the smallest average diameter of the stepped through-opening in the mixer housing is larger than the largest external diameter of the
15 working tool. This measure allows the entire working tool, including the motor, to be removed via the stepped through-opening.

Typically, both flanges have holes for fastening the flanges to the mixer housing. In this case, the larger flange can have additional openings which are preferably larger than the holes for fastening and are provided to allow a tool to access the holes or fastening means in the
20 smaller flange through the opening. This facilitates the fastening of the motor housing to the mixer housing.

In a further preferred embodiment, the tool shaft consists of two parts which are detachably fastened to each other, one part being integrally connected to the motor shaft, while the other part carries the working tool. In this case, the detachable connection can be carried out via a
25 flange connection.

Alternatively thereto, the tool shaft can also be formed in one piece with the motor shaft.

Further advantages, features and possible applications of the present invention will become clear from the following description of preferred embodiments and also from the associated figures, in which:

30 Figure 1 is a vertical section through a mixture of the prior art;
Figure 2 is a vertical section through a first embodiment according to the invention; and

Figure 3 is a vertical section through a second embodiment according to the invention.

Figure 1 shows a prior art embodiment which has already been described at the outset.

Figure 2 shows a first embodiment according to the invention. Where possible, the same reference numerals have been selected for the same parts of the mixer that have already been shown in Figure 1 and discussed. In Figure 2 the drive motor 7 is received in a motor housing 16, the motor housing 16 being fastened to the mixer cover 5 by means of two outer flanges 13, 17. It may be seen that the tool shaft 8 at its drive end functions simultaneously as the motor shaft 21. The motor shaft 21, which in the embodiment shown is embodied partly as a hollow shaft, is held by the self-aligning roller bearing 18 and also the radial bearing 19. The second outer flange 13, which is more faced toward the product space, i.e. the mixing container, has a smaller external diameter than the first outer flange 17. As a result, the entire motor can be inserted into the housing cover 5 from the outside, so that first the outer flange having the smaller external diameter is inserted into a correspondingly stepped hole in the container cover until it rests against the bottom of the extended hole. The spacing of the two outer flanges 13, 17 is designed in such a way that, in the situation shown in Figure 2, both flanges can be screwed to the housing cover 5.

If required, the motor can thus easily be detached from the housing cover and removed.

A situation of this type is shown in Figure 3 which at the same time shows a second embodiment according to the invention of the mixer. In this case, the motor, along with the working tool 6, has been detached from the housing cover 5, so that the motor, along with the working tool 6, can be removed from the corresponding opening in the container cover. The embodiment shown in Figure 3 differs from the embodiment shown in Figure 2 in that the flange connection 10 is missing, so that in this case the tool shaft and motor shaft are formed in one piece. In both embodiments shown, the axis of rotation of the working tool is arranged eccentrically to the axis of rotation of the mixing container.

The integration of the motor into a robust bearing unit for accommodating the forces and moments of the working tool produces a unit having minimal maintenance costs and the highest possible reliability. Only one shaft is guided in two bearings. This shaft takes over both the forces of the motor (for example weight forces, magnetic residual forces) and the

forces of the working tool (vortexer, kneader, etc.). Any necessary variation of the rotational speed can be facilitated by the use of a frequency converter.

List of reference numerals

	1	Mixer
	2	Mixer housing
	3	Mixing container
5	4	Ball bearing
	5	Housing cover
	6	Working tool
	7	Drive motor
	8	Tool shaft
10	9	V-belt
	10	Flange connection
	11, 12	Tool bearing
	13	Flange
	14, 15	Motor bearing
15	16	Motor housing
	17	Flange
	18	Self-aligning roller bearing
	19	Radial bearing
	20, 21	Motor shaft
20	22	Opening for assembly tool

CLAIMS

1. Mixer with a mixing container and a tool shaft arranged at least partly in the mixing container, the tool shaft having a working end to which a working tool is fastened or can be fastened and a drive end which is mounted by means of two mutually set-apart tool bearings, a drive motor being provided with a motor shaft for driving the tool shaft, wherein the motor shaft is mounted by at least one of the two mutually set-apart tool bearings.
2. Mixer according to claim 1, wherein the motor is arranged between the two tool bearings and the motor shaft is mounted preferably by means of the two tool bearings, so that the tool shaft also serves as the motor shaft.
3. Mixer according to claim 1 or 2, wherein the motor is a direct drive, preferably a three-phase synchronous motor, preferably a torque motor, servo motor or reluctance motor.
4. Mixer according to any one of claims 1 to 3, wherein one of the bearings, preferably the bearing arranged closer to the working end of the tool shaft, is a combined radial axial bearing (radiax bearing), preferably a self-aligning roller bearing or self-aligning ball bearing and particularly preferably a twin-row self-aligning roller bearing.
5. Mixer according to any one of claims 2 to 4, wherein the diameter of the motor shaft differs on the two tool bearings, preferably the diameter of the motor shaft on the tool bearing facing away from the tool shaft being smaller, preferably at least 30 % and particularly preferably at least 50 % smaller, than the diameter of the motor shaft on the other tool bearing.
6. Mixer according to any one of claims 1 to 5, wherein the motor is arranged in a motor housing, both tool bearings being arranged on or in the motor housing.
7. Mixer according to claim 6, wherein the motor housing has a first outer flange and the mixer has a mixer housing in which the mixing container is arranged, the outer flange being fastened to the mixer housing, particularly preferably to the housing cover.
8. Mixer according to claim 7, wherein the motor housing has a second outer flange which is also fastened to the mixer housing, the second outer flange having preferably a larger average diameter than the first outer flange.

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9. Mixer according to claim 8, wherein the mixer housing has a stepped through-opening with a first portion having a smaller average diameter and a second portion having a larger average diameter, the second portion having an average diameter which is larger than the average diameter of the first outer flange and is smaller than the average diameter of the second outer flange.
 10. Mixer according to claim 8, wherein the smallest stepped through-opening is larger than the largest external diameter of the working tool.
 11. Mixer according to any one of claims 1 to 10, wherein the tool shaft has two portions which are detachably fastened to each other, one of the portions being formed in one piece with the motor shaft.
 12. Mixer according to any one of Claims 1 to 10, wherein the tool shaft and the motor shaft are formed in one piece.
 13. Mixer substantially as hereinbefore described with reference to any one of the embodiments as that embodiment is shown in the accompanying drawings.

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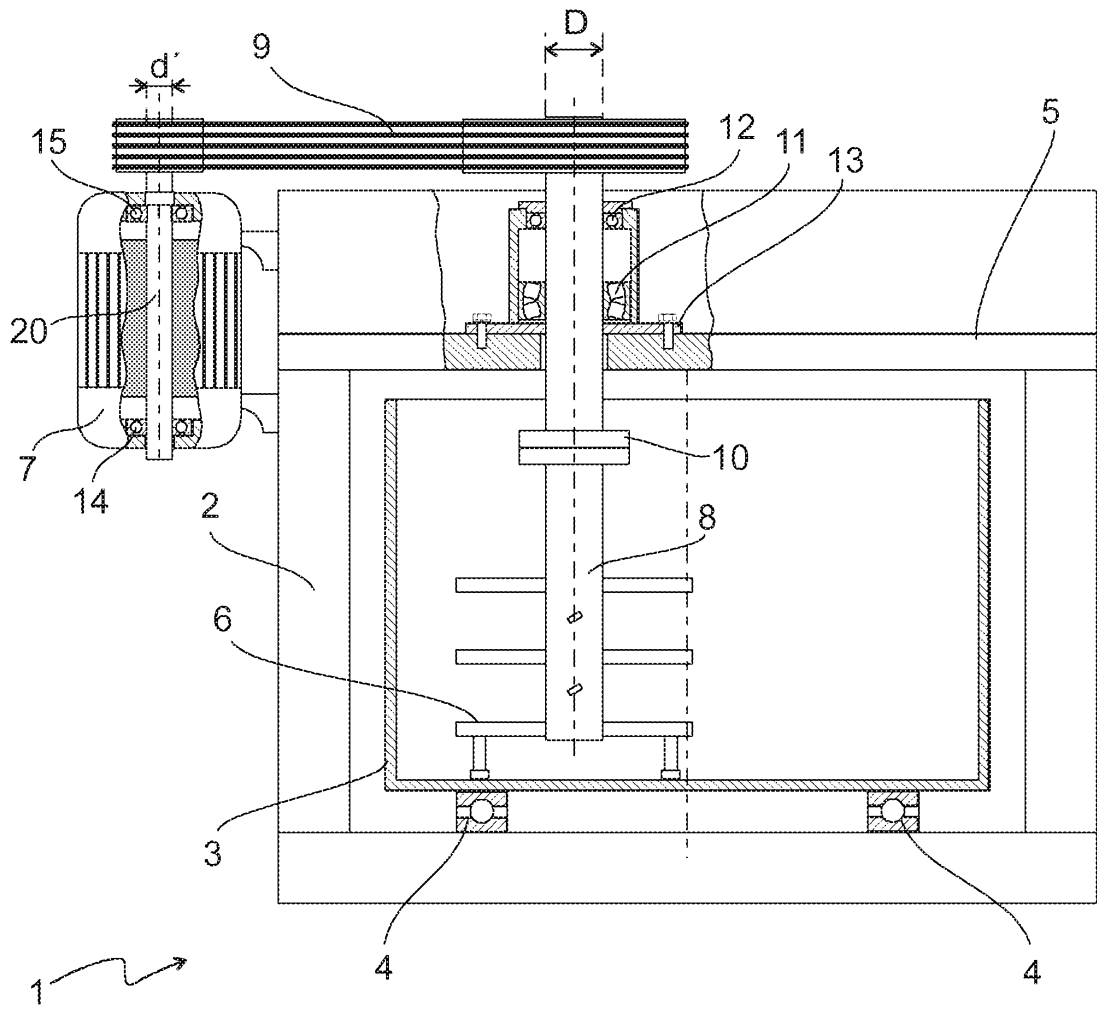


Fig. 1

