



(86) Date de dépôt PCT/PCT Filing Date: 1997/05/05

(87) Date publication PCT/PCT Publication Date: 1998/11/12

(45) Date de délivrance/Issue Date: 2002/07/23

(85) Entrée phase nationale/National Entry: 1999/10/29

(86) N° demande PCT/PCT Application No.: CA 1997/000307

(87) N° publication PCT/PCT Publication No.: 1998/050301

(51) Cl.Int.⁶/Int.Cl.⁶ B66F 11/04, B66F 1/08

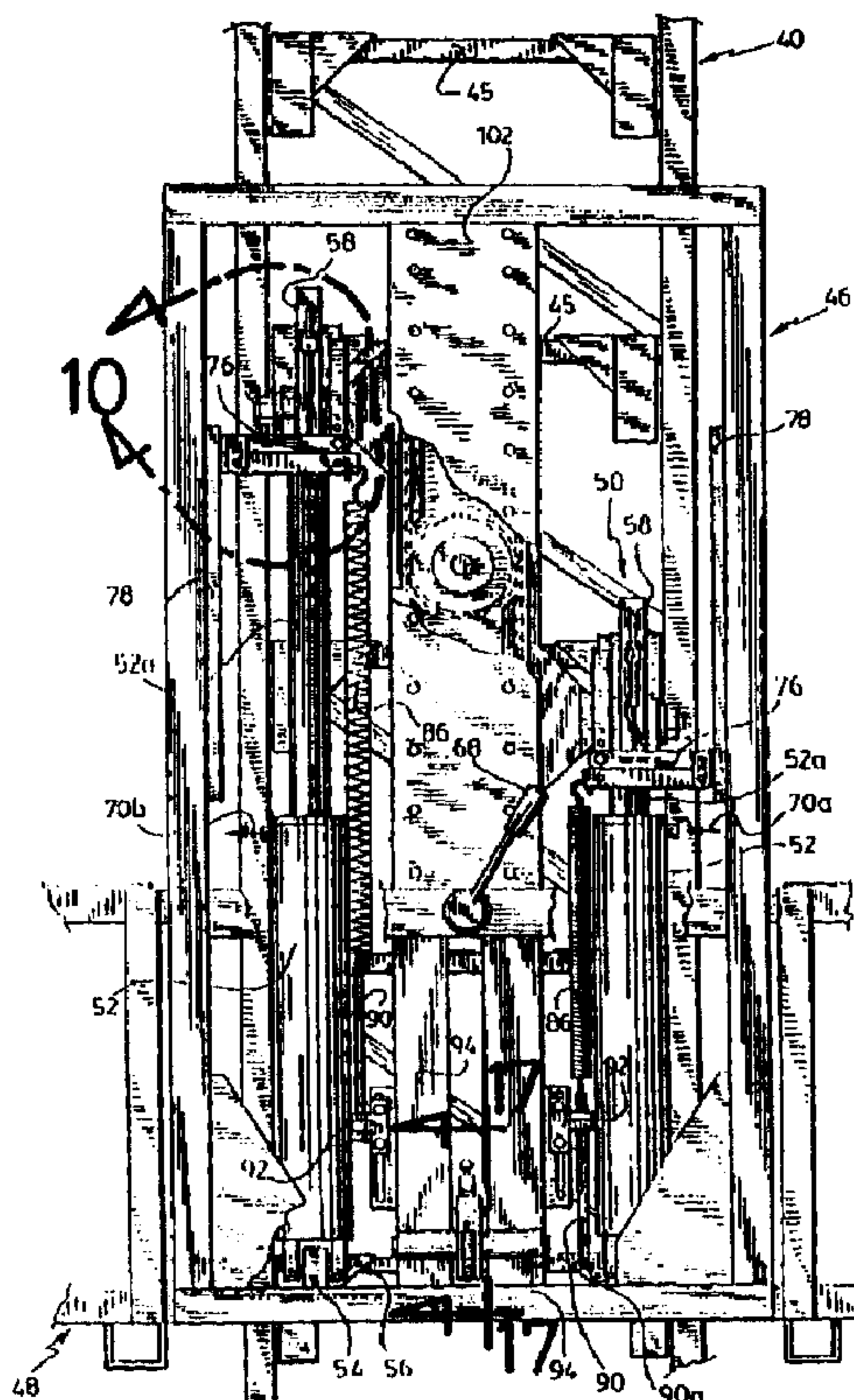
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(54) Titre : ECHAFAUDAGE A ERECTION AUTOMATIQUE

(54) Title: SELF ERECTING SCAFFOLDING



(57) Abrégé/Abstract:

A work platform (48) is supported by a sleeve (46) which is movable along a tower (40) having a plurality of vertically equally spaced rungs (45). The raising system (50) of the invention comprises two hydraulic rams (52), pivotally attached to the sleeve (46) and equipped with corresponding hooks (58) at the extremity of their piston rods (52a). The rams (52) are activated



(57) **Abrégé(suite)/Abstract(continued):**

simultaneously in up and down movement but in alternate directions, bearing in a hooking engagement in turn on the rungs (45) of the tower (40) to pull the sleeve (46) upwards along the tower (40). When a rung (45) is in the path of a hook (58) in the upward movement, the latter pivots backwards to clear the rung (45) and is pivoted back by means of springs (66). The lowering operation of the sleeve (46) is done similarly to the raising operation, but there is provided latch levers (82) that an operator must unlatch to allow springs (86) to pivot the hooks (58) in a rung clearing position during the retraction of the corresponding ram's piston rod (52a), so that the hook (58) will clear the rungs (45).

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International Bureau

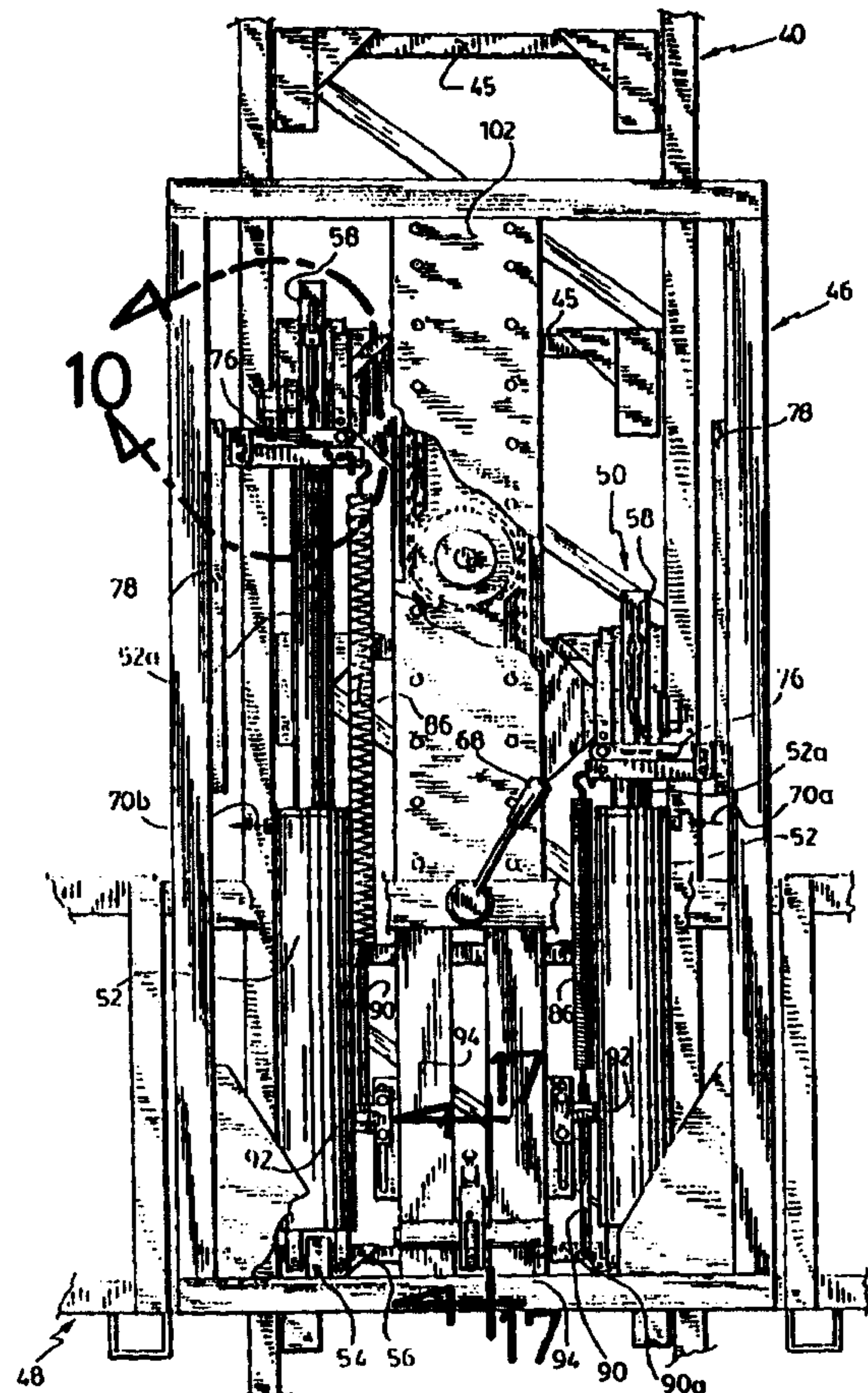
INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

(51) International Patent Classification ⁶ : B66F 11/04, 1/08	A1	(11) International Publication Number: WO 98/50301 (43) International Publication Date: 12 November 1998 (12.11.98)
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(54) Title: SELF ERECTING SCAFFOLDING

(57) Abstract

A work platform (48) is supported by a sleeve (46) which is movable along a tower (40) having a plurality of vertically equally spaced rungs (45). The raising system (50) of the invention comprises two hydraulic rams (52), pivotably attached to the sleeve (46) and equipped with corresponding hooks (58) at the extremity of their piston rods (52a). The rams (52) are activated simultaneously in up and down movement but in alternate directions, bearing in a hooking engagement in turn on the rungs (45) of the tower (40) to pull the sleeve (46) upwards along the tower (40). When a rung (45) is in the path of a hook (58) in the upward movement, the latter pivots backwards to clear the rung (45) and is pivoted back by means of springs (66). The lowering operation of the sleeve (46) is done similarly to the raising operation, but there is provided latch levers (82) that an operator must unlatch to allow springs (86) to pivot the hooks (58) in a rung clearing position during the retraction of the corresponding ram's piston rod (52a), so that the hook (58) will clear the rungs (45).



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TITLE OF THE INVENTION: SELF ERECTING SCAFFOLDINGFIELD OF THE INVENTION

5 The present invention relates to self raising platform assemblies, and more particularly to the means for raising such platforms.

BACKGROUND OF THE INVENTION

10 U.S. patent n°4,809,814 issued in 1989 to JEAN ST-GERMAIN shows a scaffolding with a platform raising system (figure 7) comprising an arm 84 and a ram member 106 that work together to raise (or lower) the platform 42. The platform 42 is vertically movable along an upstanding post 30 and is symmetrically positioned thereon. To raise
15 platform 42, ram member 106 upwardly extends its piston rod 112 which slidably and pivotably engages, with its upper end, successive transverse bars or steps 38 of post 30. A guide bar 120 is provided to slidably engage each successive step 38 and prevent piston rod 112 from abutting
20 against the lower portion of said steps. When its stroke is complete, piston rod 112 is retracted by its cylinder 108 and downwardly abuts with a hook member 118 against a step 38; therefore, when its stroke is reversed, ram member 106 raises platform 42. Arm 84 engages with a hook member
25 90 steps 38 on the opposite side of post 30, while ram member 106 is in its upward stroke, to uphold platform 42. Arm 84 is pivotable and is also equipped with a guide bar 98 to clear each successive step 38 as platform 42 is raised by ram member 106. Piston rod 112 has an upward
30 stroke allowing it to move two steps at a time, to accelerate the raising of the platform. To lower the platform, however, an operator must manually pivot arm 84 (and a second security arm 84') to allow them to clear each successive step 38.

35 U.S. patent n°5,368,125 issued in 1994 to the present applicant shows another platform raising system for raising a work platform 24 along a tower 2. Figures 9 to

- 2 -

20 show different embodiments of the raising system, though the idea is essentially the same in each embodiment. Figure 9 shows that the platform raising system comprises two power actuated hydraulic rams 36 each having a hook member 50 pivotably installed at the upper end of their piston rod. Biasing means 54, 56, 60, 62 link the hook members 50 of the two rams 36 in an opposite simultaneous pivoting movement, so that hooks 50 may pivotably engage or clear the bars or rungs of the tower frame structure (as in the JEAN ST-GERMAIN patent). To raise the work platform to which the lower part of rams 36 are attached, the latter extend their piston rod upwardly. The inclined surfaces 70 of hooks 50 slidably engage each successive rungs 10 of the tower frame structure to pivot hooks 50 and allow them to clear each of said rungs 10. A spring 62 pivots hooks 50 inwardly so that they may engage a corresponding step 10 once a rung 10 is cleared. When the downward stroke of rams 36 is initiated, hooks 50 therefore downwardly abut against, in a hooking engagement, steps 10 and the retraction of the rams' piston rods results in the raising of the work platform. During the upward stroke of rams 36, latching levers 72 abut against steps 10 to sustain the work platform (figure 9a). Levers 72 can pivot to clear each successive step 10 while they are raised with the work platform during the upward stroke of rams 36. During the lowering of the platform, an operator must manually pivot latching levers 72 to their rung unlatching position in which the platform may be lowered without hindrance to the platform lowering operation.

Both of the previously mentioned patents thus include a work platform which can be raised (or lowered) along a single tower through the instrumentality of powered rams that pull the work platform by hooking themselves on successive rungs of the tower. These patents are functional, though the raising or lowering of the platform is a relatively slow operation. Also, an operator is needed to manually pivot the arm 84 (in the JEAN ST-GERMAIN

patent) or the latching levers 72 and hook releasing handle 66 (in the ANDRE ST-GERMAIN patent) to allow the lowering of the work platform.

5 OBJECTS OF THE INVENTION

It is an object of this invention to provide a self raising system for a work platform.

It is another object of this invention that the raising system be faster than the existing systems.

10 It is yet another object of this invention that the platform lowering operation be accomplished without the constant implication of an operator.

Another object of the invention is that the system of the character described be fail-safe.

15

SUMMARY OF THE INVENTION

The present invention relates to a platform raising system in a scaffolding for raising a work platform along a tower, said platform being sustained by a sleeve member positioned around said tower, almost encircling it completely, said sleeve member being movable along said tower, said tower having a plurality of approximately parallel and equally spaced rungs, said raising system being installed on said sleeve member, said raising system comprising two rams pivotably attached to said sleeve member and having piston rods movable from a retracted position to an extracted position and at the end of which are pivotably installed corresponding hook members, said rams being connected to power means and control means that simultaneously activate each ram in up and down movement

but in alternate directions, first pivoting means for pivoting said hook members from a hooking position to an unhooking position under the action of said rams, said raising system further comprising synchronizing means for
5 ~~synchronizing the relative movement of said rams, at least~~
one of said hook members abutting against one of said rungs in said hooking position at any given time to sustain said sleeve member.

Preferably, said sleeve member is raised or
10 lowered of a distance approximately equal to the distance between two successive rungs under the action of each ram, the strokes of said rams being at least equal to slightly more than the distance between two successive rungs.

Advantageously, said first pivoting means are
15 inclined surfaces on said hook members, said inclined surfaces being slidably engageable with each successive rung to pivot said hook members in said unhooking position.

Preferably, said hook members have biasing means for pivoting them from said unhooking position to said
20 hooking position when said inclined surface is not in contact with one of said rungs.

Advantageously, said platform raising system comprises second pivoting means for pivoting said hook members from said hooking position to said unhooking
25 position, said second pivoting means comprising a latch lever member pivotable between a latched position and an unlatched position, said second pivoting means being gradually activated during the extension stroke of said rams from an inactive position when said piston rods are in
30 said retracted position, said second pivoting means pivoting one of said hook members only if: a) said hook member is not bearing on any one of said rungs in a hooking engagement; and b) said latch lever member is in said unlatched position.

35 Preferably, a latch handle is provided to manually force said latch lever member into said latched

position or into said unlatched position.

Advantageously, said second pivoting means further comprise at least one spring for each of said rams, said spring being attached to a corresponding one of said latch lever members at its first extremity and to said sleeve member at its second extremity, said spring being at its equilibrium state when said corresponding piston rod is in said retracted position.

Preferably, said hook members have biasing means for pivoting them from said unhooking position to said hooking position when said second pivoting means are not pivoting them into said unhooking position.

Advantageously, said synchronizing means are a pair of parallel toothed racks engaging peripherally a complementary gear wheel on opposite sides of the latter, each one of said racks being rigidly linked to a corresponding one of said hook members and said gear wheel being rotatably carried by said sleeve member.

Preferably, said platform raising system further comprises a security braking system, said braking system to be activated only if said sleeve member moves generally downwardly along said tower at a specific speed or faster, said specific speed being considerably faster than the speed at which the sleeve member moves along said tower during its use.

Advantageously, said braking system comprises a rocking member pivotable between a rung engaging and a rung clearing position, a retainer arm pivotable between a first and a second limit position, said retainer arm retaining said rocking member in said rung clearing position when it is in said first limit position, biasing means for pivoting said rocking member in said rung engaging position when said retainer arm is in said second limit position and a finger member for pivotable abutment against each of said successive rungs, said finger member pivoting on itself when the speed of said sleeve member is lower than said specific speed and pivoting said retainer arm from said

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5a

first limit position to said second limit position when the

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speed of said sleeve member is greater than said specific speed.

DESCRIPTION OF THE DRAWINGS

5 In the annexed drawings:

Figure 1 is a partial elevation of a tower and work platform assembly together with a platform raising system according to the invention;

10 Figure 2 is a partially fragmented elevation, at an enlarged scale, of the platform raising system of figure 1, partially showing the tower and work platform;

Figures 3 to 5 are sequential partial elevations of the tower showing the work platform and raising system at three different heights on said tower;

15 Figures 6 to 8 are cross-sectional views, at an enlarged scale, taken along lines 6-6, 7-7 and 8-8 of figures 3, 4 and 5, respectively;

20 Figure 9, on the fourth sheet of drawings, is similar to figures 6 to 8, but showing the hook member at a different position;

Figure 10 is an elevation, at an enlarged scale, taken in the area circumscribed in line 10 of figure 1;

Figure 11 is similar to figure 10, but showing the latching lever member in its unlatched position;

25 Figure 12 is a partial cross-sectional view, at an enlarged scale, taken along line 12-12 of figure 8;

Figure 13 is similar to figure 12, but showing the latching lever member in its unlatched position;

30 Figure 14 is a partial cross-sectional view taken along line 14-14 of figure 12;

Figure 15 is a partial fragmented elevation, at an enlarged scale, of the raising system of figures 1 and 2, showing more particularly the toothed racks and the gear wheel of the invention;

35 Figure 16 is a partial cross-sectional view, at an enlarged scale, taken along line 16-16 of figure 15;

Figure 17 is a partial cross-sectional view, at

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an enlarged scale, taken along line 17-17 of figure 2, showing the security braking system;

Figures 18 and 19 are similar to figure 17, but the security braking system is in different sequential positions;

Figure 20 is a cross-sectional view taken along line 20-20 of figure 17;

Figure 21 is a schematic view of the hydraulic circuit of the platform raising system of the invention;

Figure 22 is a side view of the control handle and the corresponding hydraulic connections therewith;

Figure 23 is a perspective view of the sleeve member that is destined to move along the tower, the position of the latter being suggested in dotted lines;

Figure 24 is a perspective view, at an enlarged scale, of the attachment means for the modular work platform;

Figure 25 is a top plan view of modular attachment means;

Figure 26 is a cross-sectional view taken along line 26-26 of figure 25;

Figure 27 is a cross-sectional view, at an enlarged scale, taken along line 27-27 of figure 1;

Figure 28 is a view, at an enlarged scale, of the area circumscribed in circle 28 of figure 27;

Figure 29 is a cross-sectional view taken along line 29-29 of figure 28;

Figure 30 is a partial edge view of a tower post, at an enlarged scale, showing the modular attachment means of the tower; and

Figure 31 is an exploded perspective view of the modular attachment means of figure 30.

DETAILED DESCRIPTION OF THE EMBODIMENTS

Figure 1 shows a scaffolding 38 comprising a tower 40 which rests on - and is attached to - a base 42 securingly anchored to the ground G, e.g. with bolts 44.

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Tower 40 is of the type described in the above-mentioned patent 3,368,125, except for a few improvements which will be described later, and comprises *inter alia* a plurality of generally horizontal and vertically equally spaced rungs

5 45. Tower 40 is secured to the wall (not shown) of the structure on which work is done and is spaced therefrom through the instrumentality of a plurality of wall anchors (not shown, such as the anchor members 12 described in Patent No. 3,368,125. A sleeve member 46 is movable along
10 tower 40, sleeve member 46 having a rectangular cross-section (figure 23) and being positioned around tower 40, almost encircling it completely. Sleeve member 46 has a vertical opening 47 allowing it to move along tower 40 without hindrance from the wall anchors. A work platform
15 (figure 1) 48 is removably attached to sleeve member 46 in a manner which will also be described later.

Figure 2 shows a platform raising (and lowering) system 50 according to the invention. Raising system 50 comprises two powered rams 52, 52 that are hingedly
20 attached, at the lower end of their cylinder, to sleeve member 46 through the instrumentality of ears 54 and bolts 56.

Each ram 52 has a hook member 58 (figures 2 and 6 to 9) pivotably attached at the free end of its piston
25 rod 52a by a hook bolt 59. Hook member 58 has an inclined upper surface 60.

In use, to raise work platform 48, hydraulic power means 62, shown in figure 1 (which can be, for example, an internal combustion engine coupled to an oil
30 pump), simultaneously activate each ram 52 in up and down movement but in alternate directions. The piston rod 52a of the first ram is extracted from the cylinder and the corresponding hook member 58 is moved upwardly from its initial resting position on a rung 45. Hook member 58,
35 originally in its generally vertical hooking position, slidably engages with its inclined surface 60 each successive sliding member 64 fixed to rungs 45 (figure 7),

being forced into its unhooking position. After sliding along sliding member 64, hook member 58 is biased to its hooking position through the instrumentality of a pair of hook springs 66, once it vertically clears the

5 corresponding rung 45 (figure 8). Its extension stroke finished, the first ram 52 proceeds to its retraction stroke, which will engage its hook member 58 on the rung 45 (figure 9). Since work platform 48 is hingedly fixed to rams 52, the retraction stroke of one or the other rams 52,
10 52 results in the raising of platform 48.

Since both rams 52 work simultaneously but in alternate directions, while the first ram 52 is in its extension stroke, the second ram 52 is in its retraction stroke, its hook member 58 therefore bearing on a rung 45.
15 The extension stroke of one piston rod 52a is equal to at least slightly more than the distance between two consecutive rungs 45. Consequently, since both the extension stroke of one ram 52 and the retraction stroke of the other ram 52 are simultaneous, the work platform 48
20 moves upwards of a distance approximately equal to the distance between two consecutive rungs during one ram stroke. Also, since there always is at least one hook member 58 resting on a rung 45, work platform 48 is always prevented from falling freely along tower 40. When a hook
25 member 58 bears on a rung 45 (figure 9) and that the corresponding ram 52 begins its retraction stroke, hook member 58 may pivot slightly around rung 45 due to the load sustained by ram 52. This is especially true if the rung 45 has been worn and that its corners are rounded. Figure
30 9 shows that sliding member 64 will then provide a support for the bearing surface 67 of hook member 58 and that the latter will therefore abut against sliding member 64 due to the torque induced in ram 52. This torque results from the fact that ram 52 is slightly inclined, because its lower
35 portion is pivotally attached at 59 to sleeve member 46 and its upper portion (hook member 58) rests on tower 40.

Figure 2 shows a control handle 68, linked to hydraulic power means 62, which can be positioned in three different modes of operation: a) platform raising mode; b) platform lowering mode; and c) inactive mode. The modes are explicit, and control handle 68 therefore allows an operator to control the platform raising system 50 in any of those three modes.

Figure 2 suggests with arrows 70a, 70b that hydraulic power means 62 are linked to rams 52. Hydraulic flow input arrow 70a shows where the fluid is inserted in the ram 52 that is on the verge of starting its extraction stroke, while hydraulic flow output arrow 70b shows the fluid output of the ram 52 that is about to begin its retraction stroke.

Figures 3 to 5 show three different consecutive positions of the sleeve member 46, work platform 48 and platform raising system 50 along a same section of tower 40. It can be seen that the rams 52, as previously stated, have an extension stroke equal to slightly more than the distance between two rungs 45. The upward movement of work platform 48 during this operation is substantially continuous since each step is done almost immediately after the previous step is finished. Also, since the platform may be raised in a continuous manner, the platform raising system 50 is relatively faster than the conventional "stop and go" systems.

Figures 10 to 13 show that, to lower work platform 48, there is provided a latch handle 72 pivoted around a pivot screw 74 at the first end of a slider arm 76, which is secured to the block fastened to piston rod 52a and carrying pivot bolt 59 of hook member 58 (see Figure 8 and 12). Slider arm 76 is slidably guided at its second end in a rail 78 (figure 2) which is fixedly anchored to sleeve member 46. Slider arm 76 can therefore freely slide vertically along rail 78 to follow corresponding hook

member 58 during each stroke of ram 52.

When latch handle 72 is pivoted downwardly by an operator, as suggested in figures 11 and 13, a U-shaped link 80 pivots a latch lever member 82 generally downwardly around a pivot pin 84 from its latched position to its unlatched position. Indeed, figure 14 shows that the first and second legs 80a and 80b of U-shaped link 80 are rotatably attached to latch handle 72 and to latch lever member 82, respectively. First leg 80a is attached non-coaxially relative to pivot screw 74 on latch handle 72 so as to provoke the generally downward movement of its second leg 80b around pivot pin 84, therefore forcing latch lever member 82 into its unlatched position

If there is no pressure applied on latch handle 72 by an operator, the position of latch lever member 82 will depend on the state of a pair of springs 86, 86. Figures 2 and 6 show that springs 86 are attached at their upper end to a spring plate 88 (fig. 6) and at their lower end to a respective piston 90 which is slidably and vertically movable inside a bored piston guide 92 fixedly but adjustably attached to a vertical beam 94 of sleeve member 46 structure (Figure 2).

If latch handle 72 is not pivoted downwardly, link 80 will keep latch lever member 82 in its latched position (figures 10 and 12).

Figures 10 to 13 show a wire 96 adjustably attached, e.g. by means of a set screw 98, at its first extremity to hook member 58 and passing through bores in slider arm 76 and in latch lever member 82. A retainer cap 100 is fixed at the second extremity of wire 96, under latch lever member 82. Since hook springs 66 tend to keep hook member 58 in its hooking position, wire 96 will pull on retainer cap 100 and contribute to keep latch lever member 82 in its latched position. When hook member 58 pivots into its unhooking position (figure 7), since link 80 keeps latch lever member 82 in its latched position, wire 96 will be free to move downwardly under the (less

than considerable) weight of retainer cap 100.

However, if latch handle 72 is pivoted downwardly, latch lever member 82 will be free to move from its latched position to its unlatched position. When the

5 piston rod 52a of corresponding ram 52 is retracted (as the right hand ram 52 of figure 2), there is no tension in spring 86 and hook member 58 will be kept in its hooking position by hook springs 66, therefore forcing with wire 96 latch lever member 82 into its latched position. When
10 piston rod 52a is gradually extracted, hook member 58 moves upwardly and piston 90 will slide correspondingly upwards in piston guide 92, until its lower end equipped with a radial flange 90a abuts against piston guide 92, near the one third of the course of piston rod 52a. For the rest of
15 the piston rod extraction, spring 86 will be gradually stretched and it will apply a downward force on latch lever member 82, thus applying a pressure to pivot hook member 58 into its unhooking position through the instrumentality of wire 96.

20 Therefore, when latch handle 72 is kept in its upwardly oriented position, springs 86 have no effect whatsoever. But when latch handle 72 is pivoted downwardly, hook member 58 will be pivoted into its unhooking position unless it already engages a rung 45
25 which would prevent this. Indeed, figure 9 shows that the retaining surface 101 (by abutting on rung 45) of hook member 58 prevents the latter from pivoting to its unhooking position. Thus, when the operator desires work platform 48 to move downwardly, he pivots both latch
30 handles 72, 72 downwardly and he positions control handle 68 to its platform lowering mode. When a ram 52 engages with its hook member 58 a rung 45, the hook member 58 of the other ram 52 still engages the next upper rung 45 and, an instant later, as this second ram 52 extracts its piston
35 rod 52a even more, hook member 58 vertically clears this rung 45 and is free to pivot to its unhooking position under the action of corresponding springs 86, 86. Ram 52

is then free to proceed to the retraction of its piston rod, since hook member 58 will then not hinder its downward movement because hook member 58 is in its unhooking position which clears rungs 45. When spring 86 ceases to
5 apply tension because it is unextended (near the one third of its course and after clearing two consecutive rungs 45), hook member 58 regains its hooking position under the bias of springs 66 well before it engages a rung 45. The
10 downward movement of work platform 48 is therefore accomplished without the constant intervention of the operator.

To move work platform 48 upwards, the operator must pivot both latch handles 72 into their upwardly oriented position (figure 10) and position control handle
15 68 in its platform raising mode.

Figure 2 shows that vertical beams 94 hold a gear housing 102. Figure 15 shows that gear housing 102 houses two slidable toothed racks 104, 104, which are fixedly attached (e.g. welded) by means of a rack plate 106 to a
20 corresponding hook bolt 59, and a complementary gear wheel 108. Rack plates 106 have a generally triangular shape and have flanges 106a extending along racks 104 to allow bolting of one to the other. Housing 102 has openings on its sides to allow rack plates 106 to extend outwardly
25 therefrom (figure 16).

The racks 104 and gear wheel 108 assembly synchronizes the relative movement of hook members 58, 58 to prevent them from being unattuned with one another, which would of course mean that the platform raising system
30 50 would be inoperable. Both hook members 58 therefore always stay in registration with one another.

Figures 17 to 20 show a security braking system 110 installed between the two vertical beams 94 of sleeve member 46 (figure 2). Braking system 110 comprises a
35 rocking member 112 which can pivot around a holding pin 114 between a rung clearing position (figure 17) and a rung engaging position (figure 19). Rocking member 112 has a

1133 1133 1133 1133

13a

sectorial edge 115 and a blocking arm 116 having a T-shaped

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cross-section at its free extremity 116a (figure 20) and extending opposite sectorial edge 115 on rocking member 112, T-shaped extremity 116a being a counterweight for the rest of rocking member 112. A concentric arcuate slot 118
5 is provided near sectorial edge 115 through which a blocking pin 120 is slidable. The rung engaging and rung clearing positions are defined by the corresponding limit positions of rocking member 112 sliding around blocking pin 120 between the two positions in which the extremities of
10 arcuate slot 118 abuts against blocking pin 120.

Braking system 110 further comprises a retainer arm 122 positioned over rocking arm 112 and pivotable around an arm pin 124. Retainer arm 122 has a wheel 126 near arm pin 124 which engages a complementary widthwise
15 groove 128 in rocking member 112 when it is in the rung clearing position. A finger 130 is pivotably installed on retainer arm 122 generally opposite wheel 126 relative to arm pin 124, a finger spring 132 also linking it to retainer arm 122.

20 A tension spring 134 is hooked to a hook 135 protruding from sectorial edge 115 of rocking member 112 at its first extremity and to a spring pin 136 fixed to vertical beams 94, 94 at its second extremity. Tension spring 134 is stretched when rocking member 112 is in its
25 rung clearing position, and lengthwisely paritally bears on sectorial edge 115 of rocking member 112.

In use, finger spring 132 is in its equilibrium state when finger 130 is between two rungs 45 (figure 17 in full lines), the free extremity of finger 130 then
30 extending beyond the vertical plane of rungs 45 shown in figures 17 to 19 by axis 138: finger 130 is then in its rung engaging position. When sleeve member 46 is lowered at a normal (relatively slow) rate, finger 130 will hit smoothly each successive rung 45 and pivot upwardly until
35 rung 45 is cleared (figure 17 in dotted lines). Finger spring 132 will pivot finger 130 back into its rung engaging position afterwards. When sleeve member 46 is

raised, finger 130 will hit smoothly each successive rung 45 and pivot downwardly until rung 45 is cleared (figure 17 in dotted lines). Again, finger spring 132 will pivot finger 130 back in its rung engaging position afterwards.

5 If sleeve member 46 was to descend abnormally fast (i.e. falling) as suggested in figures 18 and 19, finger 130 would hit the next rung 45 at high speed, relative to the usual lowering rate, and it would tilt backwards suddenly. The force of this sudden backward
10 movement would be transmitted to retainer arm 130 which would in turn pivot counterclockwisely as suggested in figures 18 and 19 causing wheel 126 to escape from groove 128 and allow rotation of rocking arm 112 around holding pin 114. Tension spring 136 would then pivot rocking arm
15 112, as suggested with an arrow in figure 18, from its rung clearing position until the latter reaches its rung engaging position (figure 19). Once wheel 126 disengages groove 128, the inertia of counterweight free extremity 116a of blocking arm 116 will also contribute to pivot
20 rocking arm 112.

When the next rung 45 is encountered, blocking arm 116, now in the rung engaging position and therefore extending beyond plane 138, will abut against this rung 45 (figure 19) and stop the fall of sleeve member 46.

25 Figure 21 shows the hydraulic circuit 140 of platform raising system 50. There is provided a motor 142 (internal combustion or otherwise) linked to a pump 144 which feeds circuit 140 with the proper fluid (e.g. oil) from a supply reservoir 146. The fluid will be fed
30 simultaneously to each ram 52, but the hydraulic flow input will be fed alternately to the upper and lower circuit connections 52b and 52c, respectively, of rams 52, which will result, as is known in the art, in the alternate movement of the piston rods 52a of rams 52. A flow
35 controller 148 will determine, through the instrumentality of control handle 68 (figure 22), the direction of the

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flow, which in turn will determine whether the sleeve member is raised or lowered.

A basic form of sleeve member 46 is shown in figure 23. This sleeve member has a central sleeve 150 and one generally cubic side structure 152 on each side of central sleeve 150. Each side structure 152 has a plurality of joists and cross bars, as is detailed in the previously cited patents. Under each side structure 152 are fixedly attached rectangular tubes 153 for slidably inserting complementary rectangular support rods (not shown). On these support rods a platform can be removably installed for supporting workmen. A second and a third platform (not shown) can also be installed on sleeve member 46, on the top bars 154 of side structures 152, for supporting equipment used by the workmen at a height that would be easily reachable without having to bend down.

To the basic sleeve member 46 can be added a plurality of structural module members 156 (figures 24 to 26) that are similar to the cubic side structures 152, 152. Figures 23 to 26 show that these module members can be removably installed on the outer end of side structures 152, 152 by means of hooking members 158 slidably engageable into complementary hook casings 160. Any number of these module members 156 can be added to the basic sleeve member 46 (up to the structural capacity limit of the material being used, of course) since every module member 156 is equipped with hooking members 158 on one side and with hook casings 160 on the other.

It is understood that sleeve member 46 and all module members 156 attached thereto must be of approximately equally distributed weight and approximately symmetrically installed relative to tower 40, so as to be generally balanced and not induce important stresses in tower 40 due to lateral tilting of the latter.

Figure 27 shows the tower 40 and sleeve member 46 engagement. It can be seen in figures 27 to 29 that idle rollers 162 are installed on the inner side of sleeve

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member 46 to rotatably abut against the outer side of tower 40 on its corner tower posts 163 to prevent sleeve member 46 from moving in any direction except along tower 40.

Figures 29 to 31 show that tower 40 is modular.

5 Indeed, attachment means 164 are provided to fixedly attach two tower modules 166 to one another. With this modular tower 40, the height of the tower can be easily adjusted by the workmen. Attachment means 164 can be, for example, of the type shown in figures 30 and 31, comprising a slider
10 plate 168 slidably engaging a pair of parallel receiving plates 170, 170 and fixed thereto by means of a bolt 172.

WE CLAIM:

1. A platform raising system in a scaffolding for raising a work platform (48) along a tower (40), said platform being sustained by a sleeve member (46) positioned around said tower, almost encircling it completely, said sleeve member being movable along said tower, said tower having a plurality of approximately parallel and equally spaced rungs (45), said raising system (50) comprising two rams (52) attached to said sleeve member (46) and having piston rods movable from a retracted position to an extracted position and at the end of which are pivotably installed corresponding hook members (58), said rams (52) being connected to power means (142, 144) and control means (148), first pivoting means (60) for pivoting said hook members (58) from a hooking position to an unhooking position under an extension stroke of said rams (52), said raising system (50) being characterized in that said power means and control means simultaneously activate each ram in up and down movement but in alternate directions and in that it further comprises synchronizing means (104, 108) for synchronizing the relative movement of said rams, at least one of said hook members (58) in hooking engagement with one of said rungs (45) in said hooking position at any given time to sustain said sleeve member (46), second pivoting means (96, 82, 86) for pivoting said hook members (58) from said hooking position to said unhooking position, said second pivoting means comprising a latch lever member (82) pivotable between a latched position and an unlatched position, said second pivoting means being gradually activated during the extension stroke of said rams from an inactive position when said piston rods are in said retracted position, said second pivoting means pivoting one of said hook members to said unhooking position only if: a) said hook member is not in hooking engagement with any one of said rungs; and b) said latch lever member is in said unlatched position.

2. A platform raising system as defined in claim 1, wherein said first pivoting means are inclined surfaces (60) on said hook members (58), said inclined surfaces being slidably engageable with each successive rung (45) to pivot said hook members to said unhooking position, and further including biasing means (66) biasing said hook members to pivot from said unhooking position to said hooking position

3. A platform raising system as defined in claim 1, further including a latch handle (72) connected to said latch lever member (82) to manually force the latter into said latched position or into said unlatched position.

4. A platform raising system as defined in claim 1, wherein said second pivoting means further comprise at least one tension spring (86) for each of said rams (52), said spring (86) being attached to a corresponding one of said latch lever members (82) at its first extremity and to said sleeve member (46) at its second extremity, said spring (86) being at its unstretched state when said corresponding piston rod is in said retracted position.

5. A platform raising system as defined in claim 2, further including a latch handle (72) connected to said latch lever member (82) to manually force the latter into said latched position or into said unlatched position.

6. A platform raising system as defined in claim 5, wherein said second pivoting means further comprise at least one tension spring (86) for each of said rams (52), said spring (86) being attached to a corresponding one of said latch lever members (82) at its first extremity and to said sleeve member (46) at its second extremity, said spring (86) being at its unstretched state when said corresponding piston rod is in said retracted position.

7. A platform raising system as defined in claim 1, wherein said synchronizing means are a pair of parallel toothed racks (104) engaging peripherally a complementary gear wheel (108) on opposite sides of the latter, each one
5 of said racks being rigidly linked to a corresponding one of said hook members, said gear wheel rotatably carried by said sleeve member 46.

8. A platform raising system as defined in claim
10 7, further including a latch handle (72) connected to said latch lever member (82) to manually force the latter into said latched position or into said unlatched position.

9. A platform raising system as defined in claim
15 8, wherein said second pivoting means further comprise at least one tension spring (86) for each of said rams (52), said spring (86) being attached to a corresponding one of said latch lever members (82) at its first extremity and to said sleeve member (46) at its second extremity, said
20 spring (86) being at its unstretched state when said corresponding piston rod is in said retracted position.

10. A platform raising system as defined in claim 1, further comprising a security braking system
25 (110), said braking system to be activated only if said sleeve member (46) moves generally downwardly along said tower at a specific speed or faster, said specific speed being considerably faster than the speed at which the sleeve member moves along said tower during its use.

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11. A platform raising system as defined in claim 10, wherein said braking system (110) comprises a rocking member (112) pivotable between a rung engaging and a rung clearing position, a retainer arm (122) pivotable
35 between a first and a second limit position, said retainer arm (122) retaining said rocking member (112) in said rung clearing position when it is in said first limit position,

biasing means (134) for pivoting said rocking member (112) in said rung engaging position when said retainer arm is in said second limit position and a finger member (130) for pivotable abutment against each of said successive rungs, said finger member (130) pivoting on itself when the speed of said sleeve member is lower than said specific speed and pivoting said retainer arm (122) from said first limit position to said second limit position when the speed of said sleeve member (46) is greater than said specific speed.

12. A platform raising system as defined in claim 9, further comprising a security braking system (110), said braking system to be activated only if said sleeve member (46) moves generally downwardly along said tower at a specific speed or faster, said specific speed being considerably faster than the speed at which the sleeve member moves along said tower during its use.

13. A platform raising system as defined in claim 12, wherein said braking system (110) comprises a rocking member (112) pivotable between a rung engaging and a rung clearing position, a retainer arm (122) pivotable between a first and a second limit position, said retainer arm (122) retaining said rocking member (112) in said rung clearing position when it is in said first limit position, biasing means (134) for pivoting said rocking member (112) in said rung engaging position when said retainer arm is in said second limit position and a finger member (130) for pivotable abutment against each of said successive rungs, said finger member (130) pivoting on itself when the speed of said sleeve member is lower than said specific speed and pivoting said retainer arm (122) from said first limit position to said second limit position when the speed of said sleeve member (46) is greater than said specific speed.

14. A scaffolding comprising:

a) a vertical tower (40) having a plurality of substantially horizontal and vertically equally spaced rungs (45);

5 b) a work platform (48) vertically movable along said tower;

c) a sleeve member (46) partly surrounding and vertically movable along said tower (40), said work platform secured to said tower;

10 d) a first and a second rams (52) pivotably attached to said sleeve member, each said ram comprising an extendable piston rod (52a) having an outer free end and a pivotable hook member (58) attached at said outer free end, at least one of said hook members resting on one of said
15 rungs at any given time to sustain said work platform;

e) first pivoting means (60) for pivoting said hook members from a hooking position to an unhooking position under the action of said rams;

f) power means (144) for activating said rams, said
20 scaffolding being characterized in that said power means alternately activates said rams and in that it further comprises;

g) synchronizing means (104, 108), for synchronizing the movement of the piston rods into simultaneous and
25 opposite action thereof;

wherein said work platform can be raised or lowered along said tower in a continuous movement through the action of said rams being alternately activated, for alternate engagement of said hook members on said rungs.

30

15. A scaffolding as defined in claim 14, wherein said synchronizing means are a pair of parallel toothed racks (104) engaging peripherally a complementary gear wheel (108) on opposite sides of the latter, each one
35 of said racks being rigidly linked to a corresponding one of said hook members, said gear wheel rotatably carried by said sleeve member 46.

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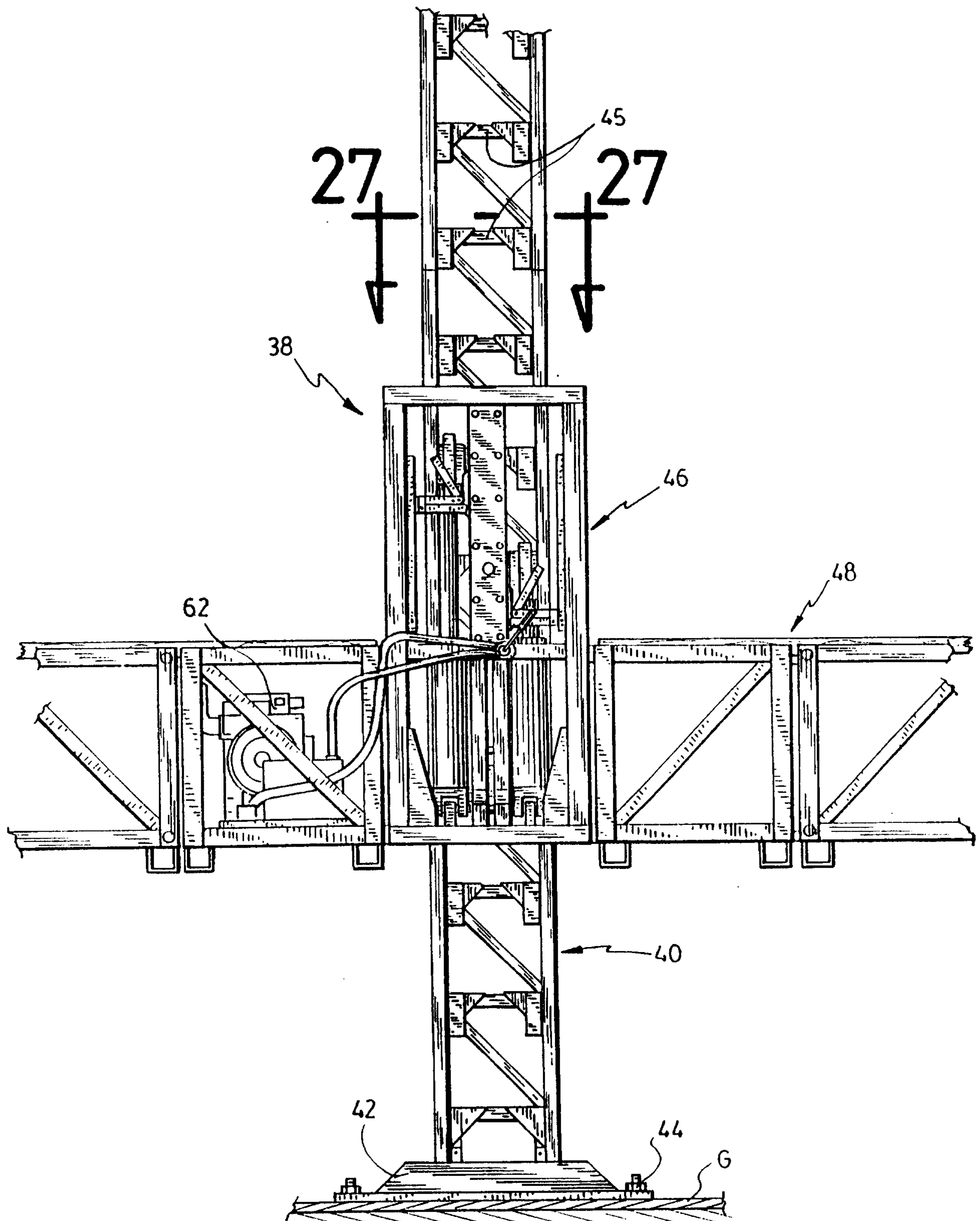
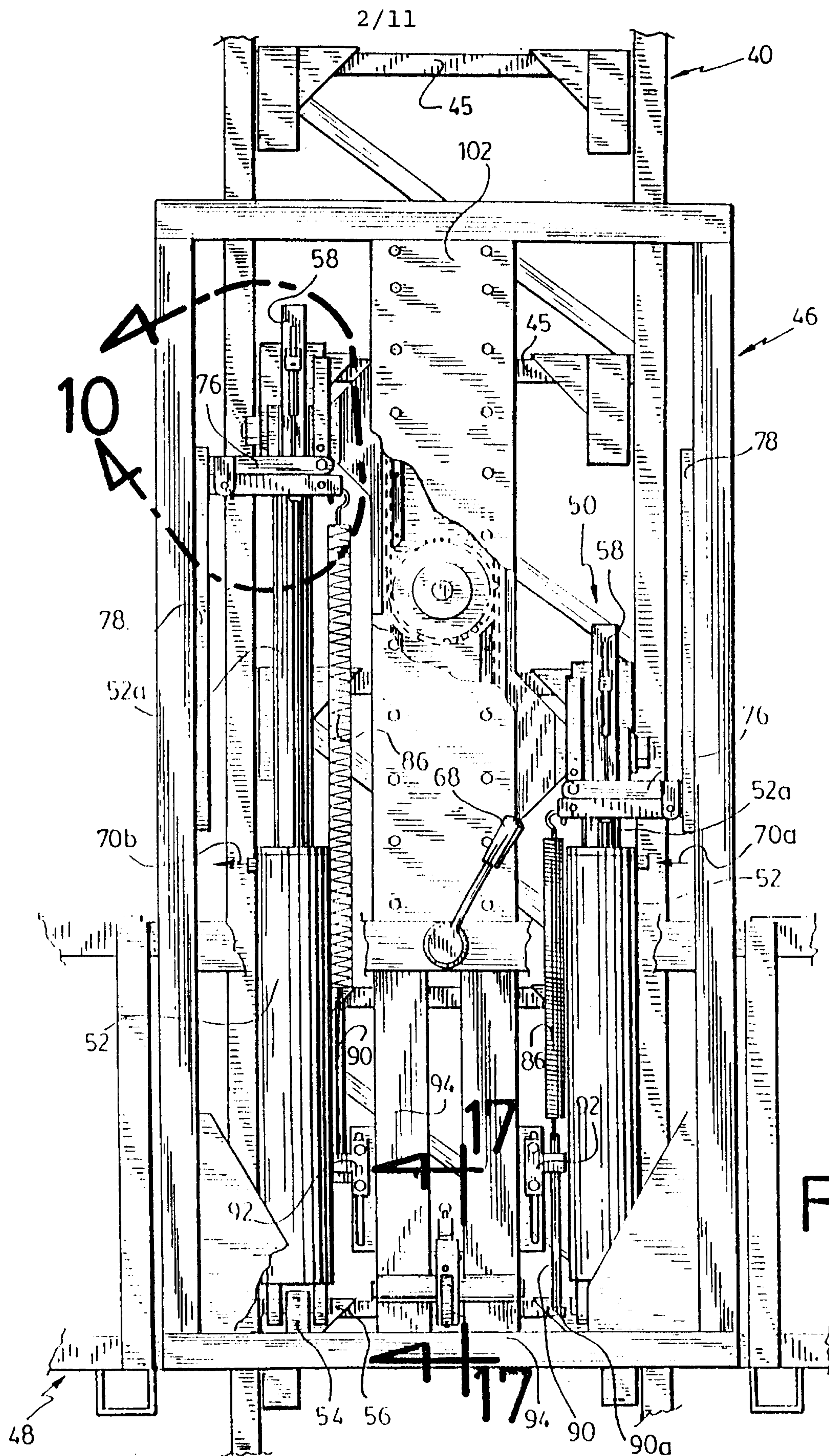


Fig.1



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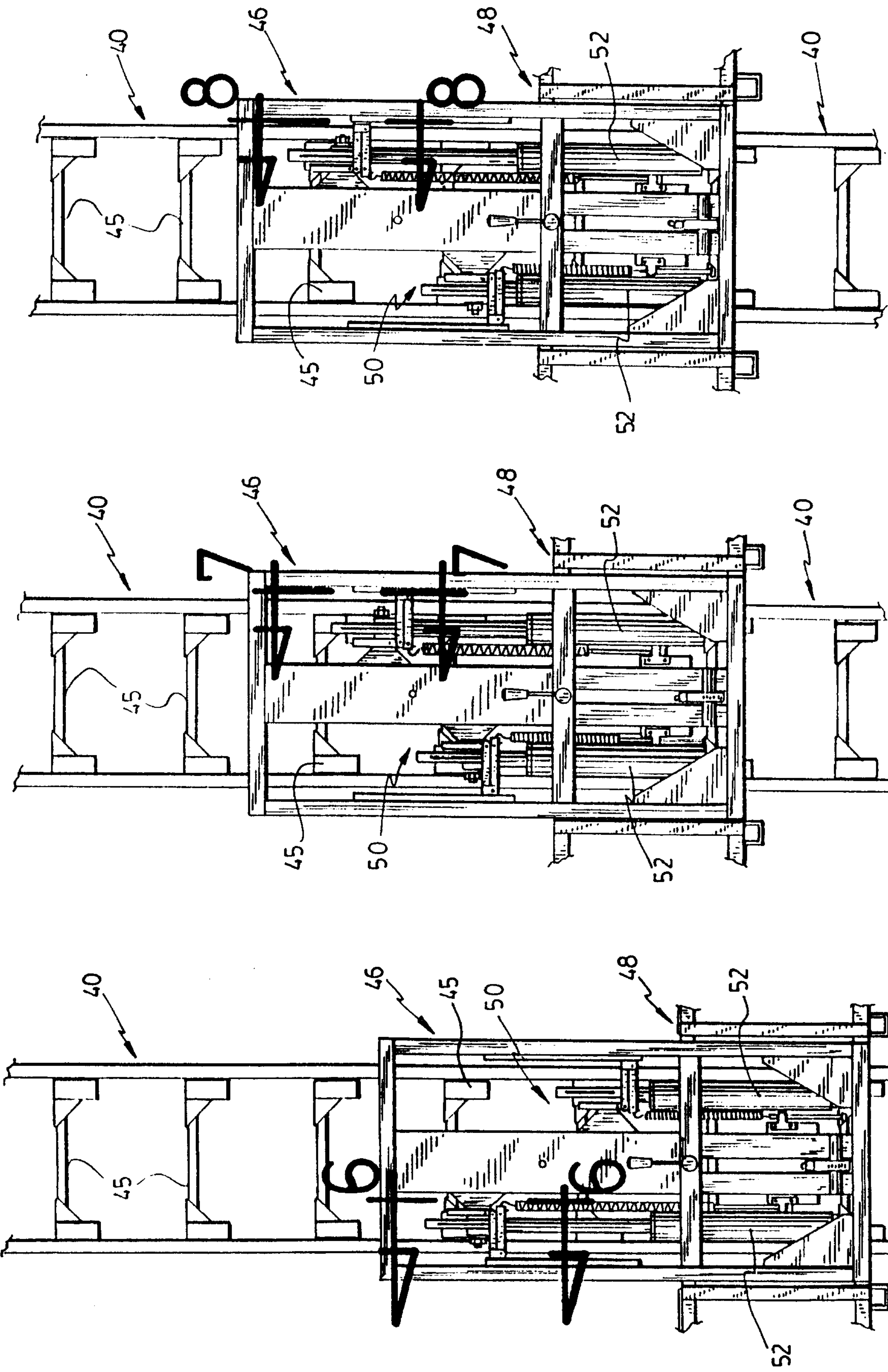
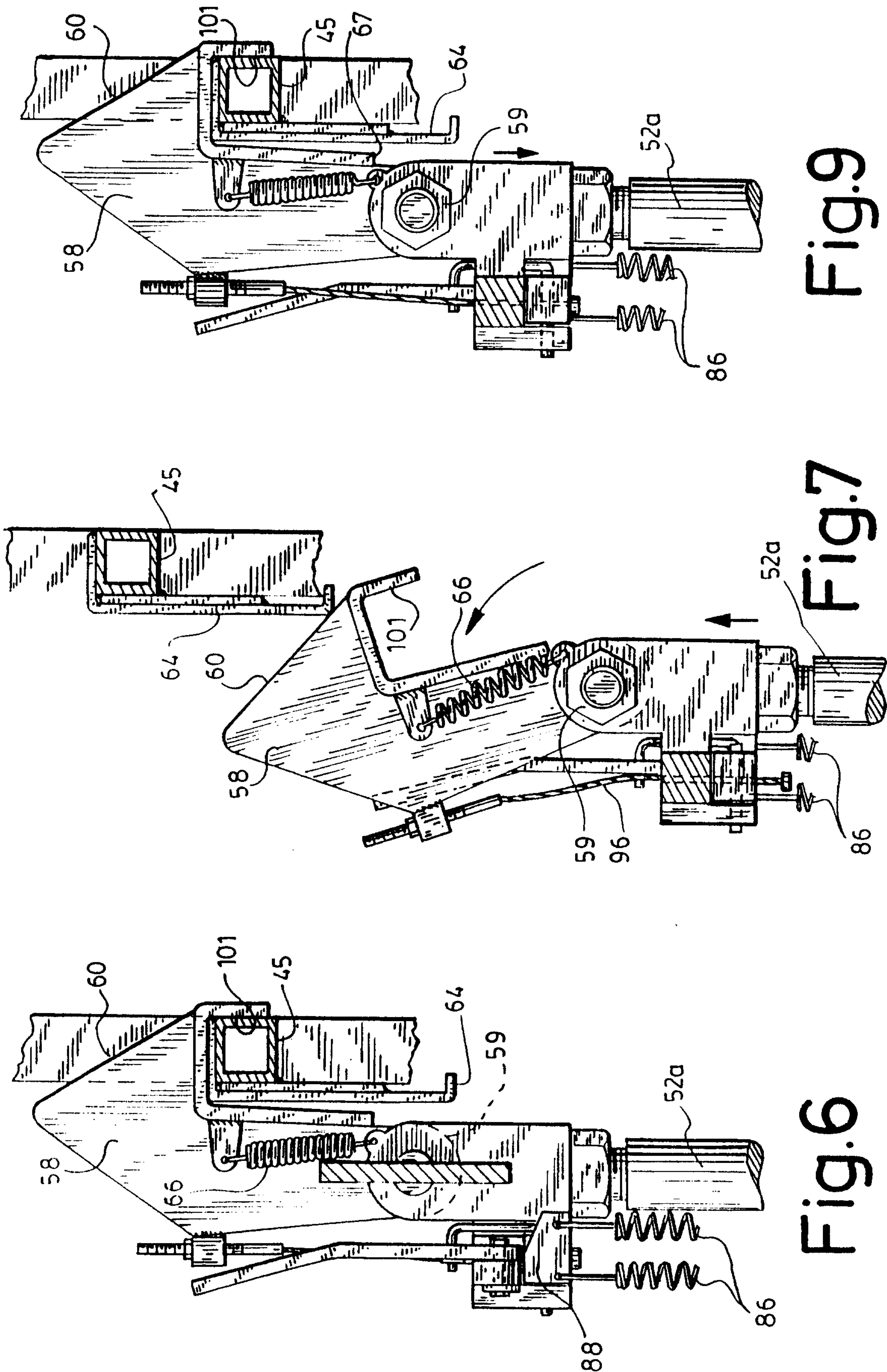


Fig.5

Fig.4

Fig.3

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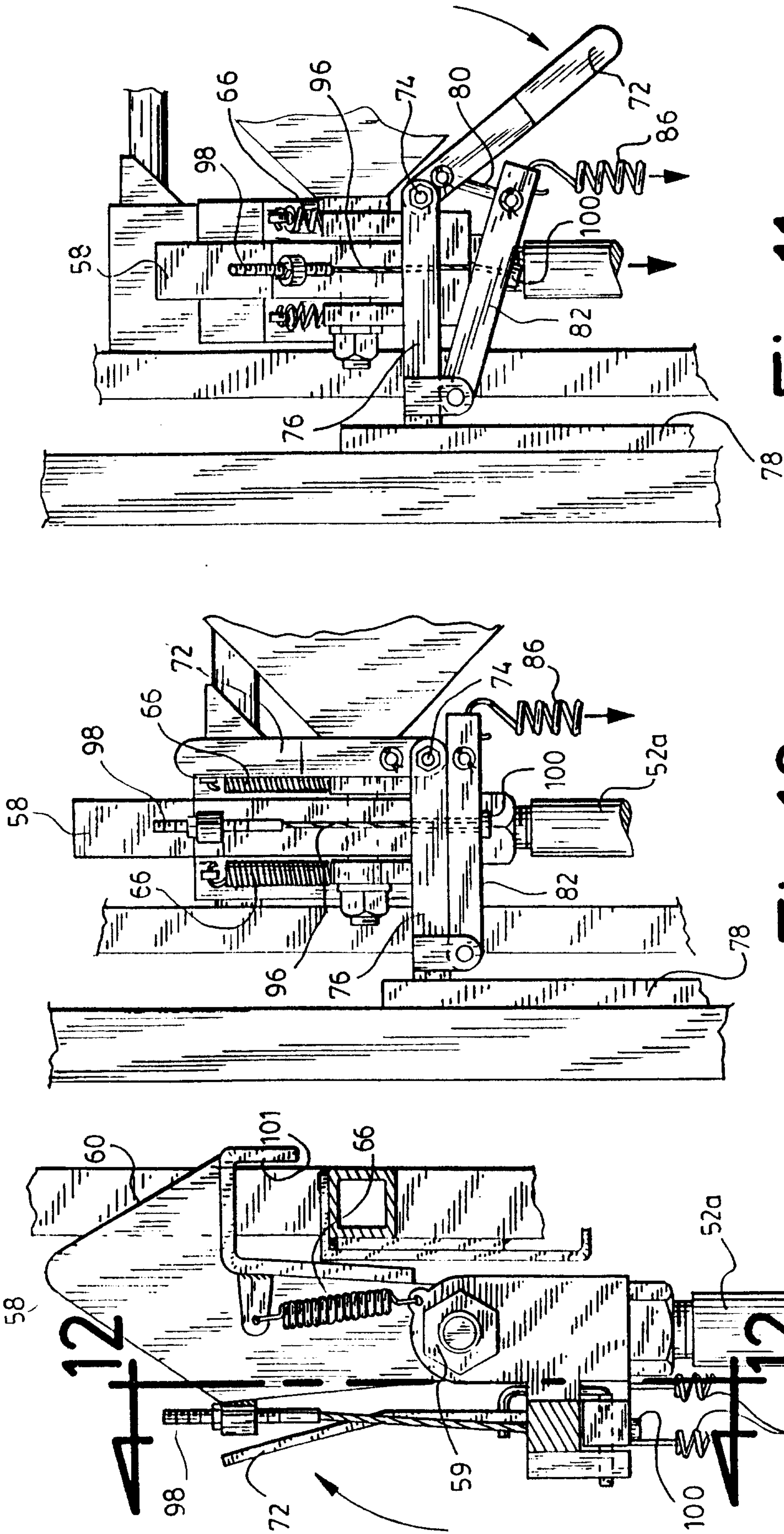


Fig.11

Fig.10

Fig.8

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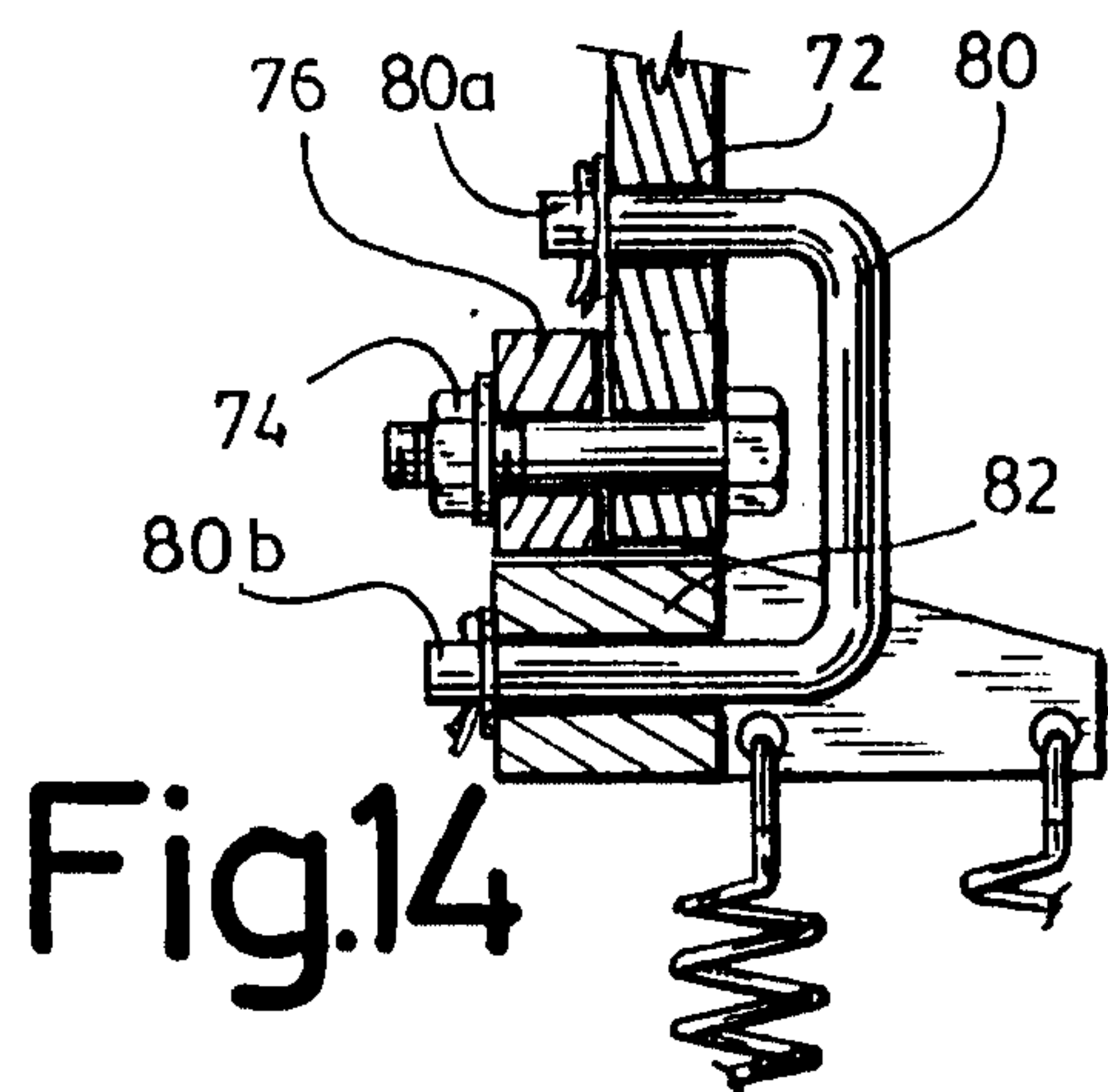


Fig.14

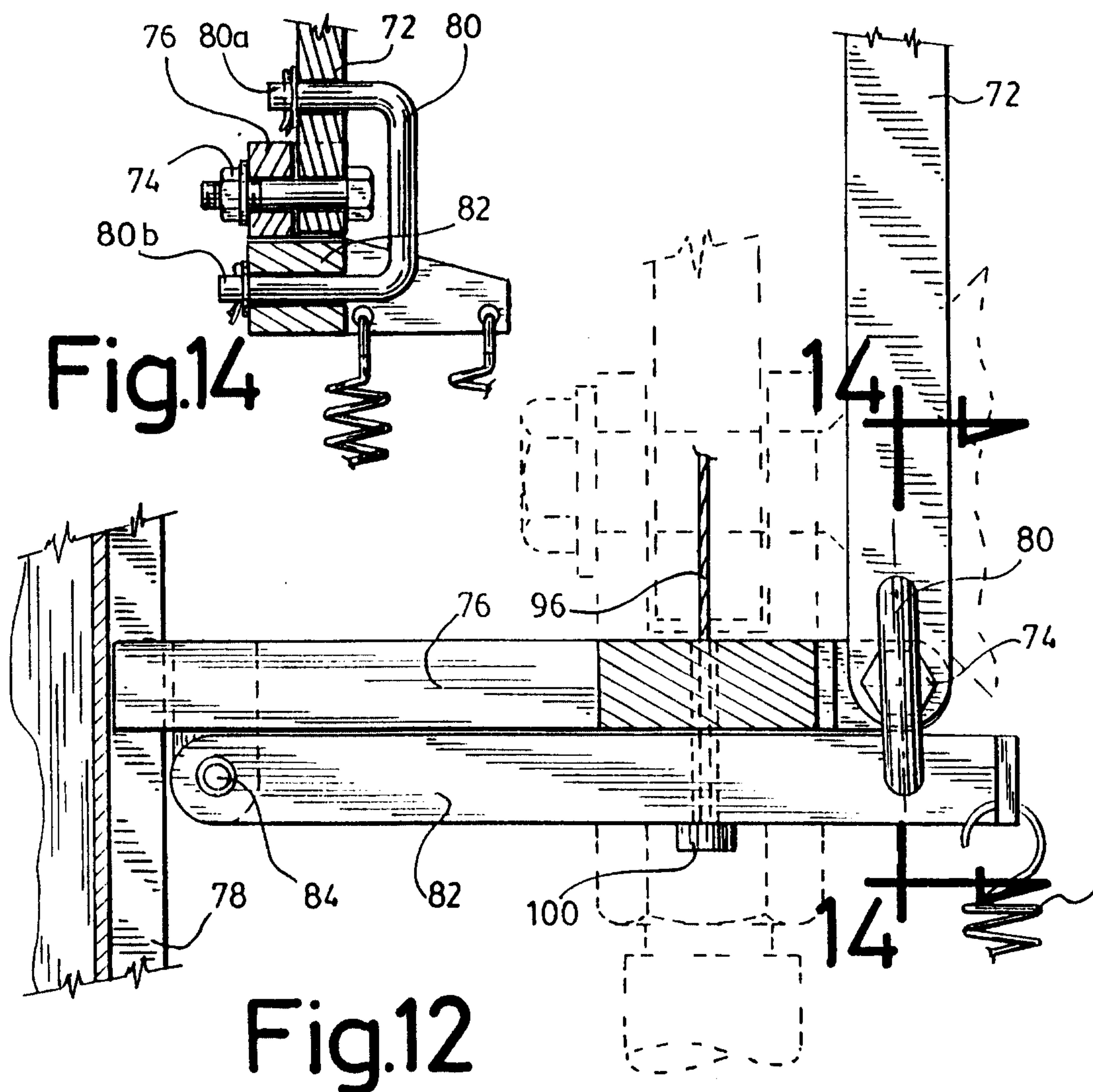


Fig.12

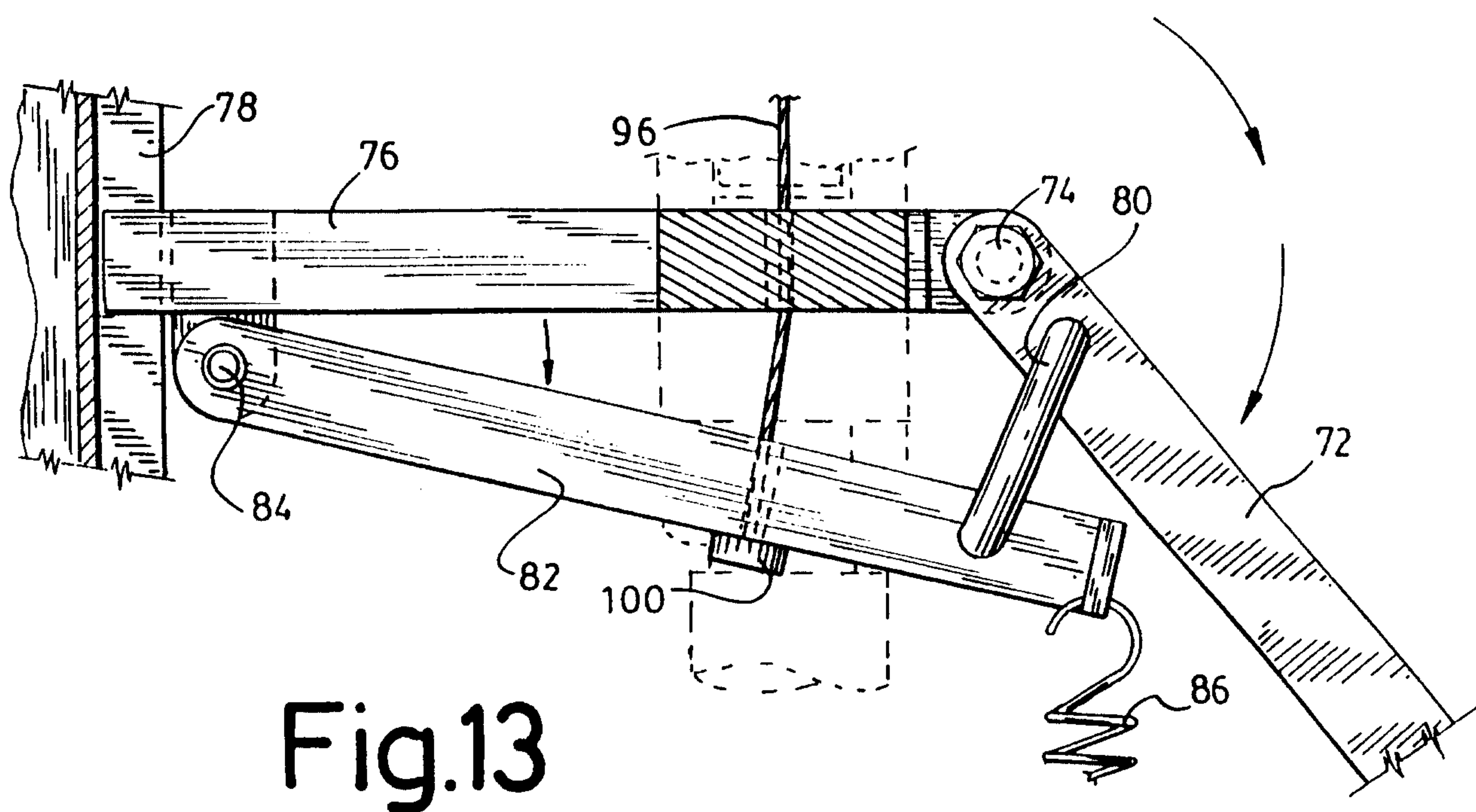
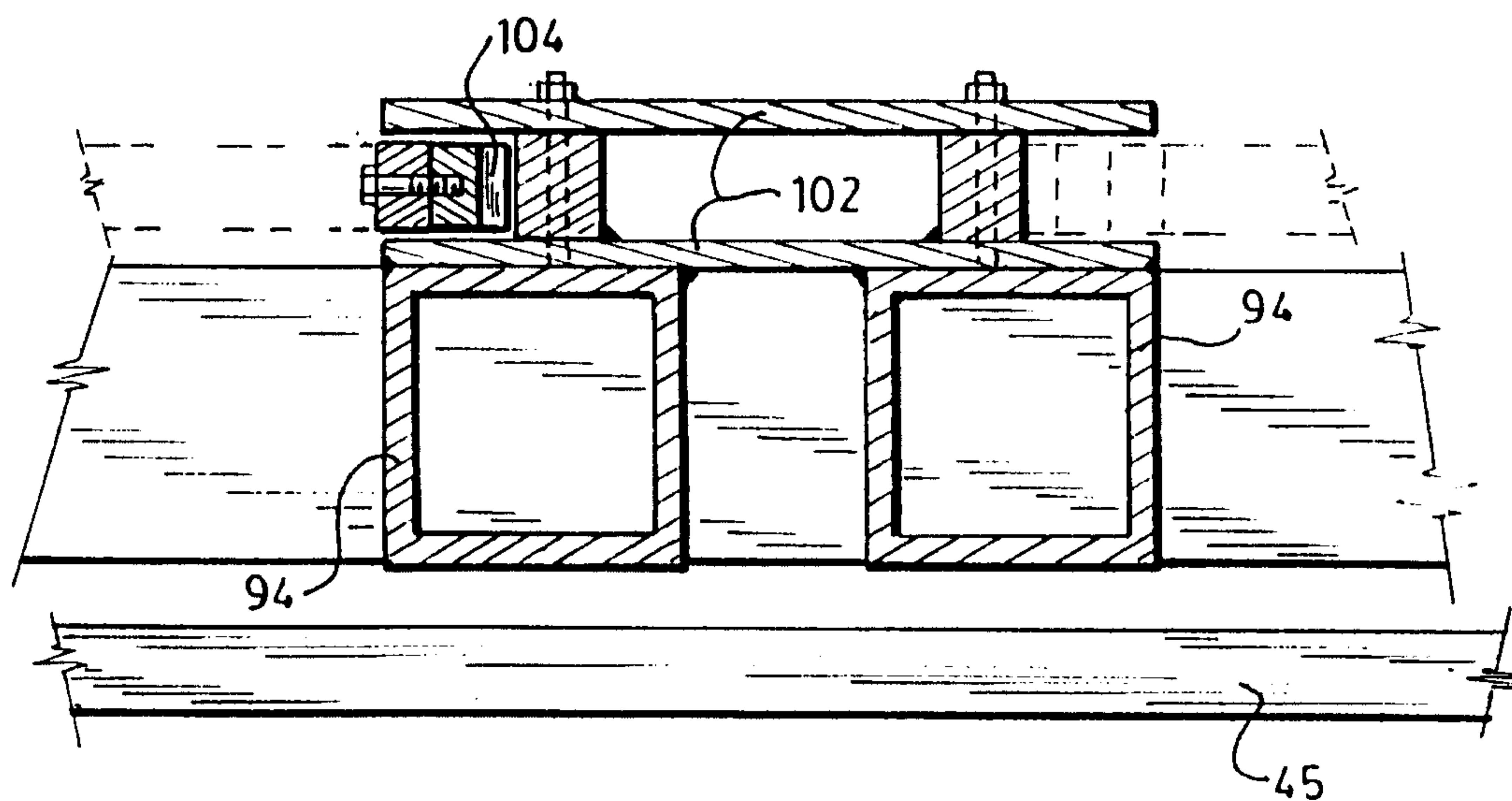
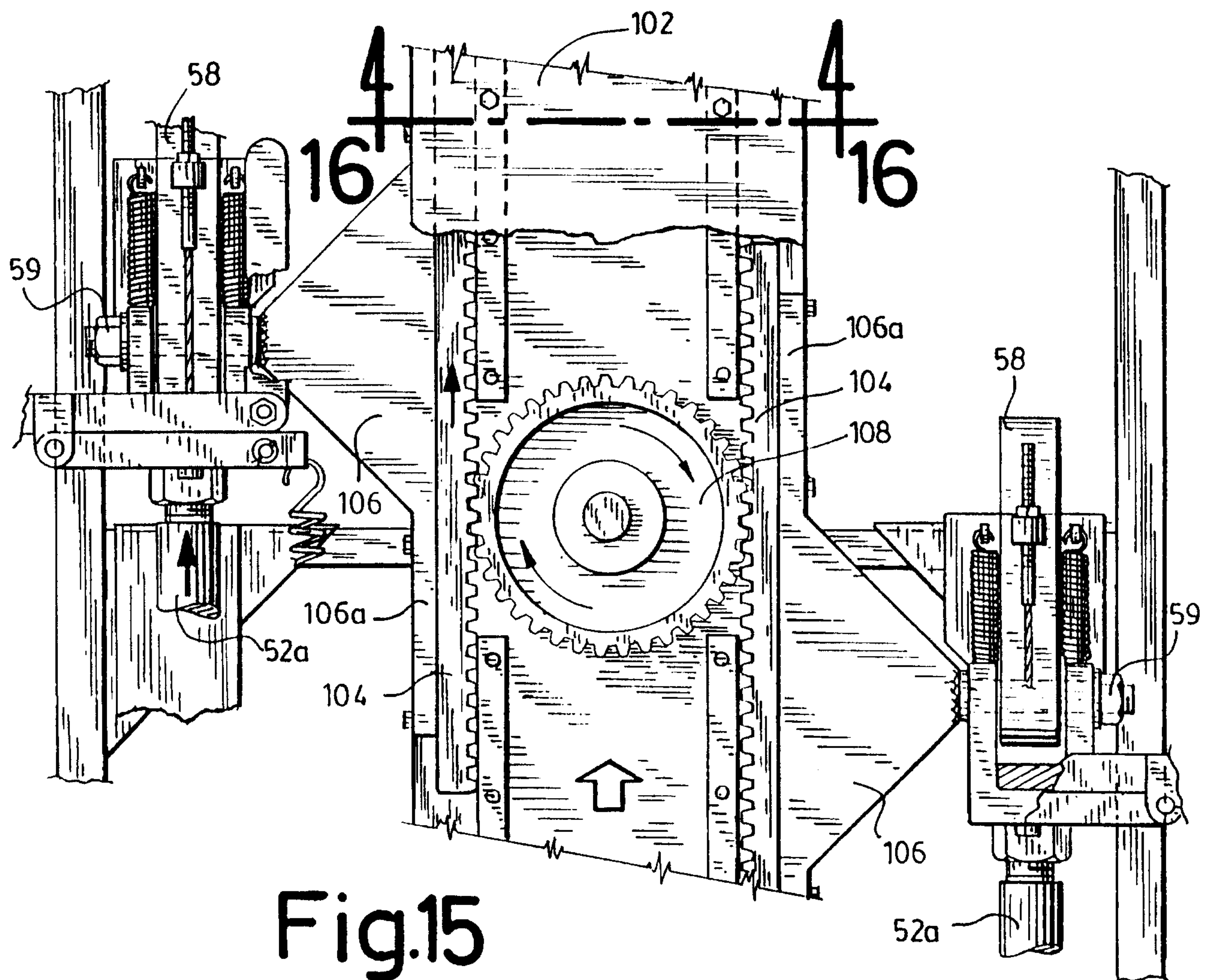


Fig.13

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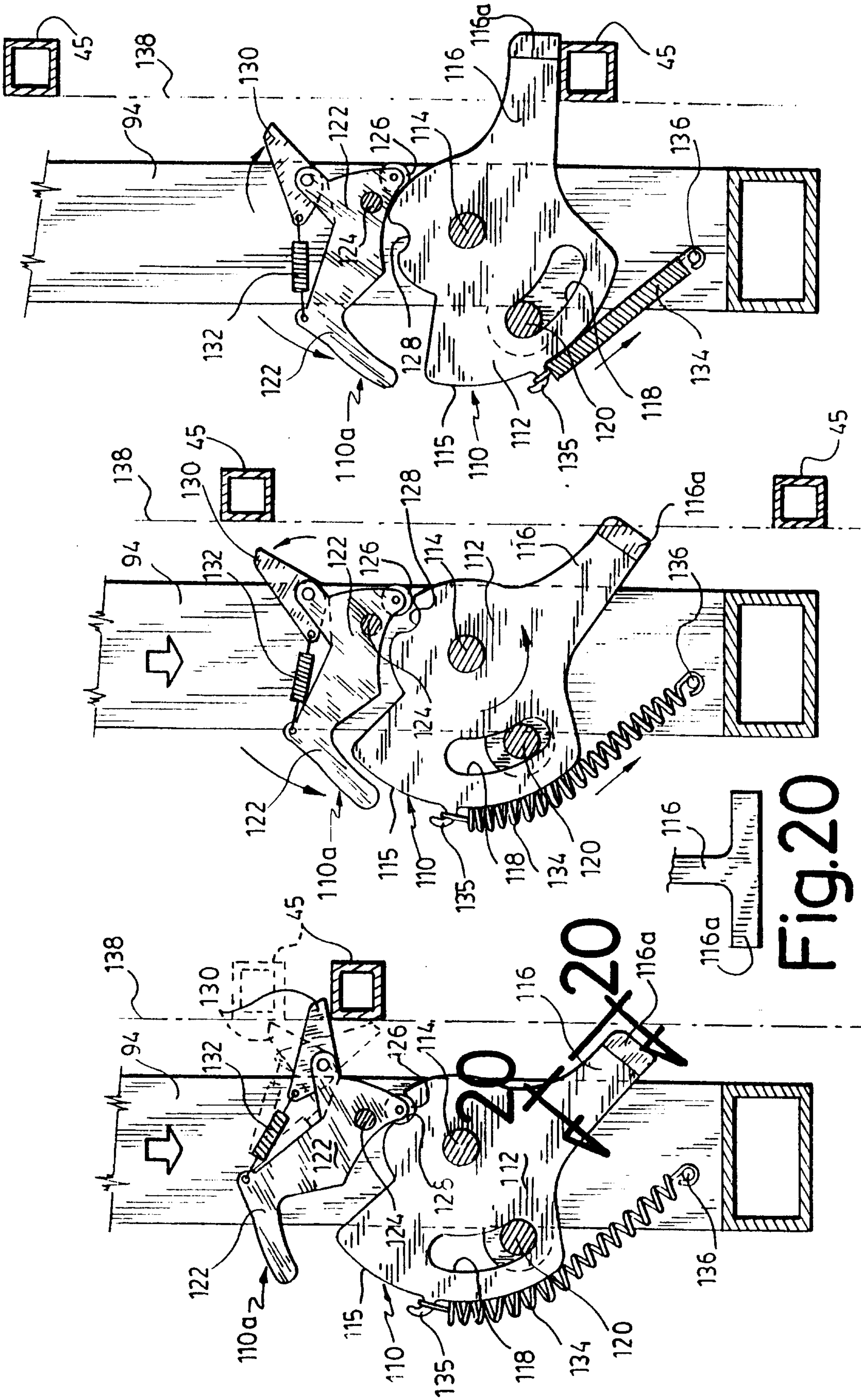


Fig.17

Fig.18

Fig.19

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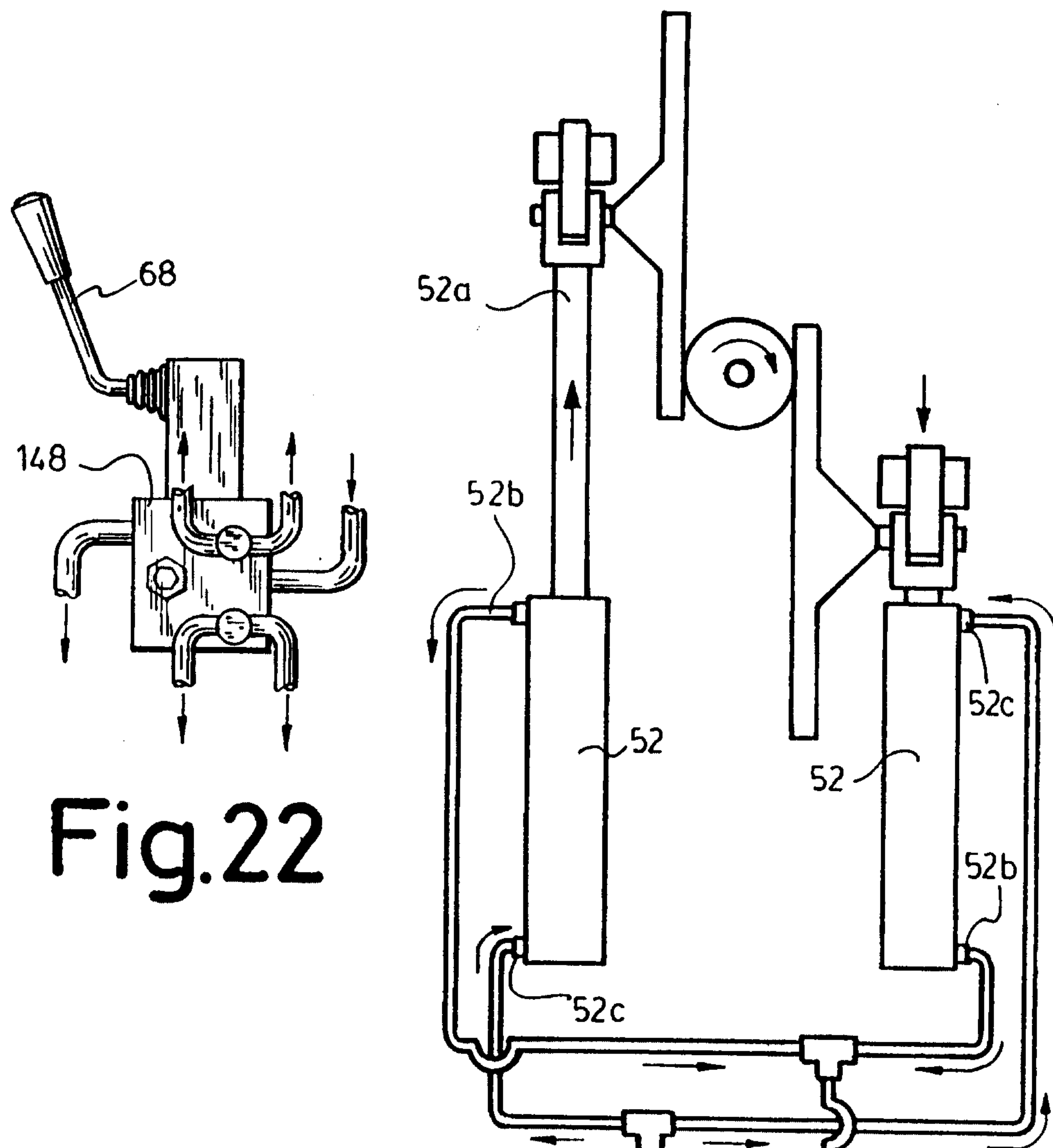


Fig.22

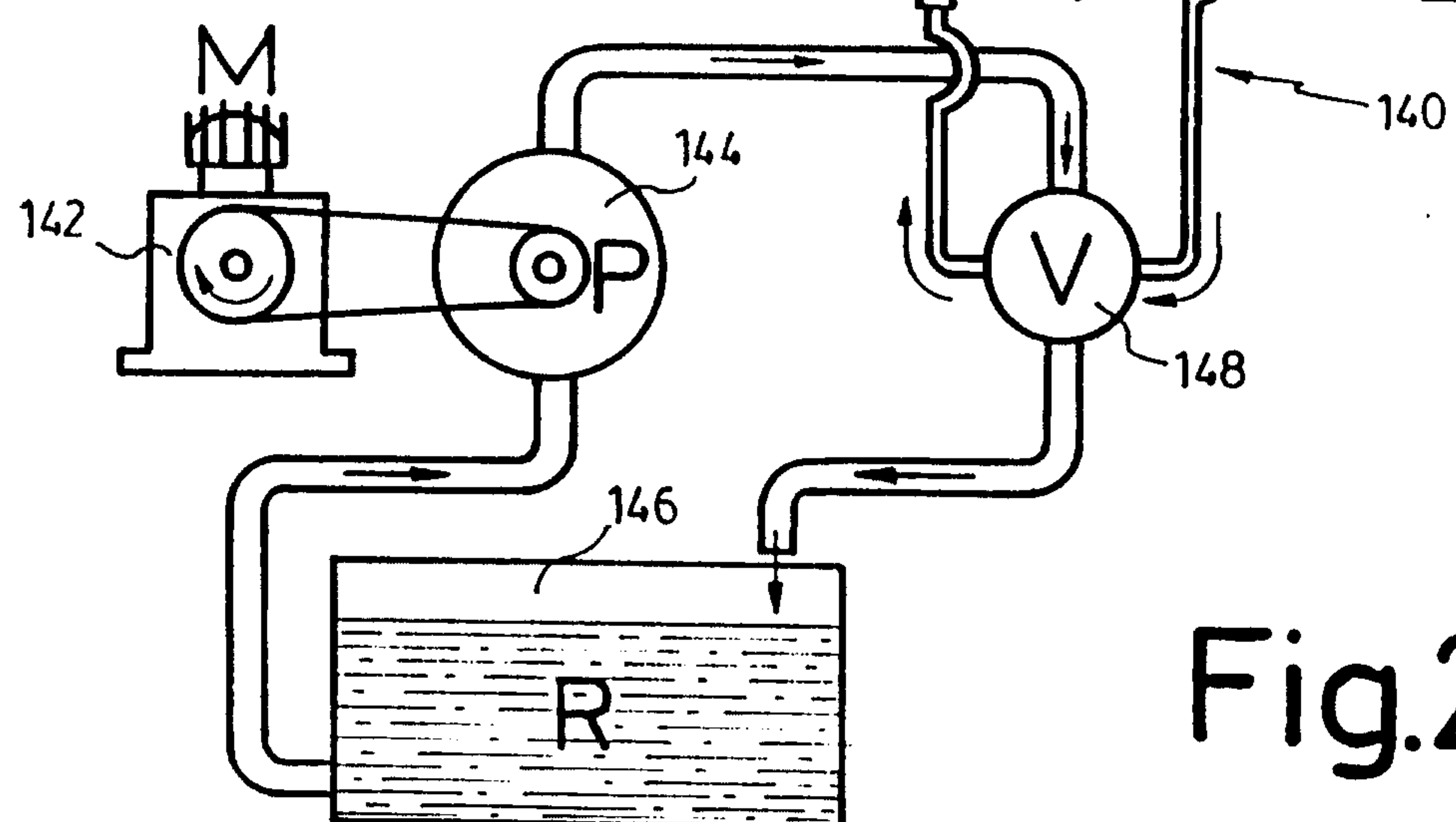


Fig.21

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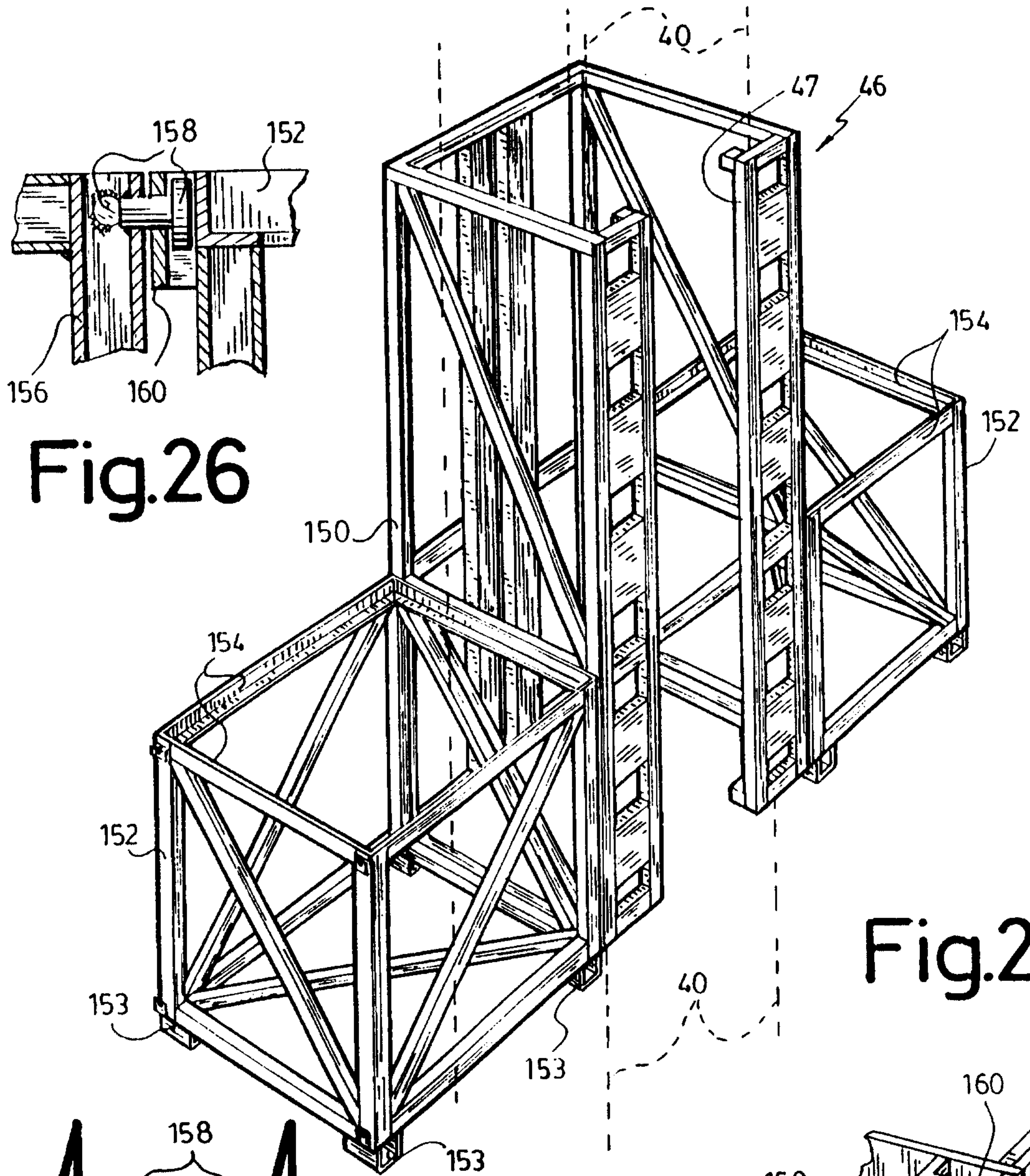


Fig.26

Fig.23

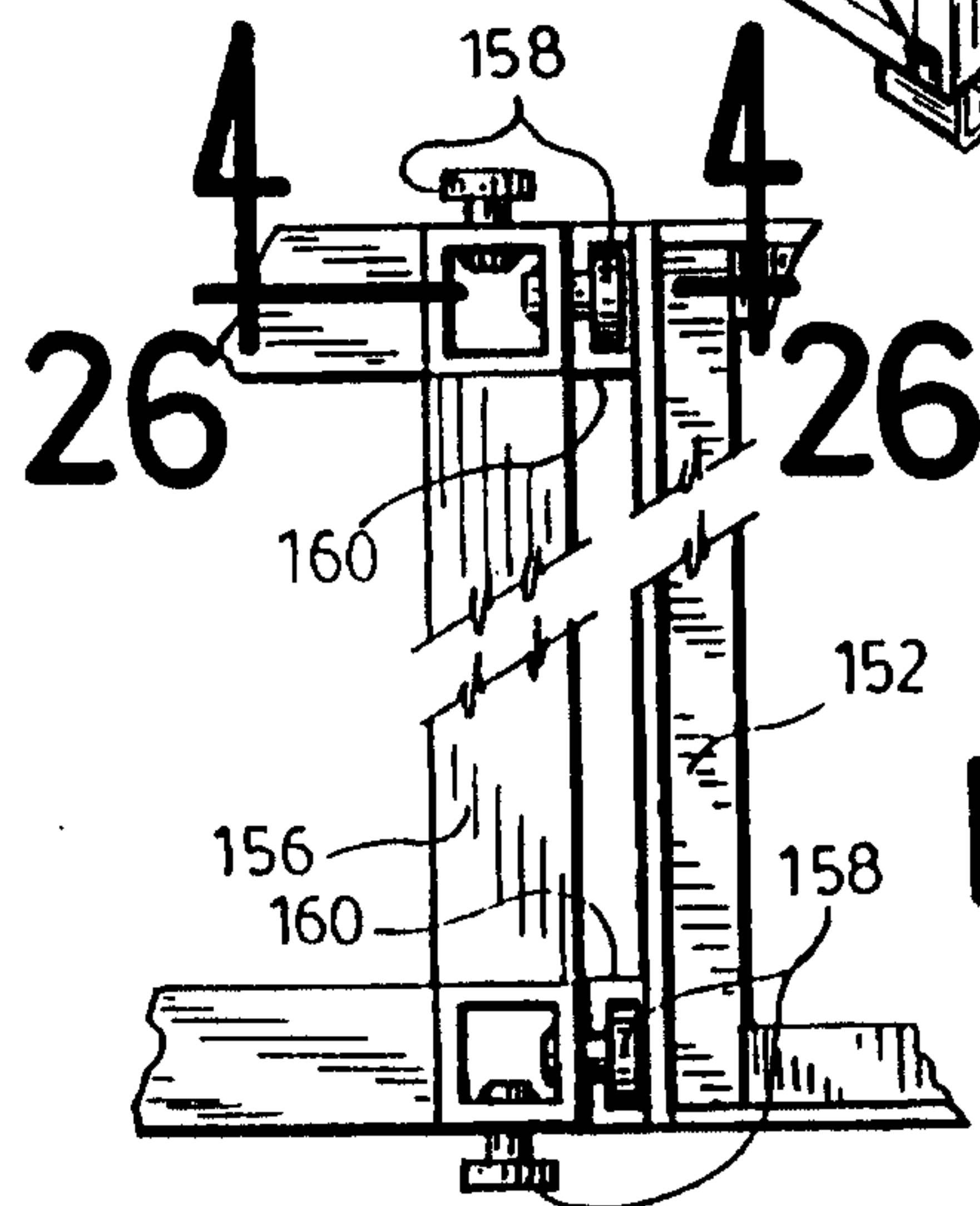


Fig.25

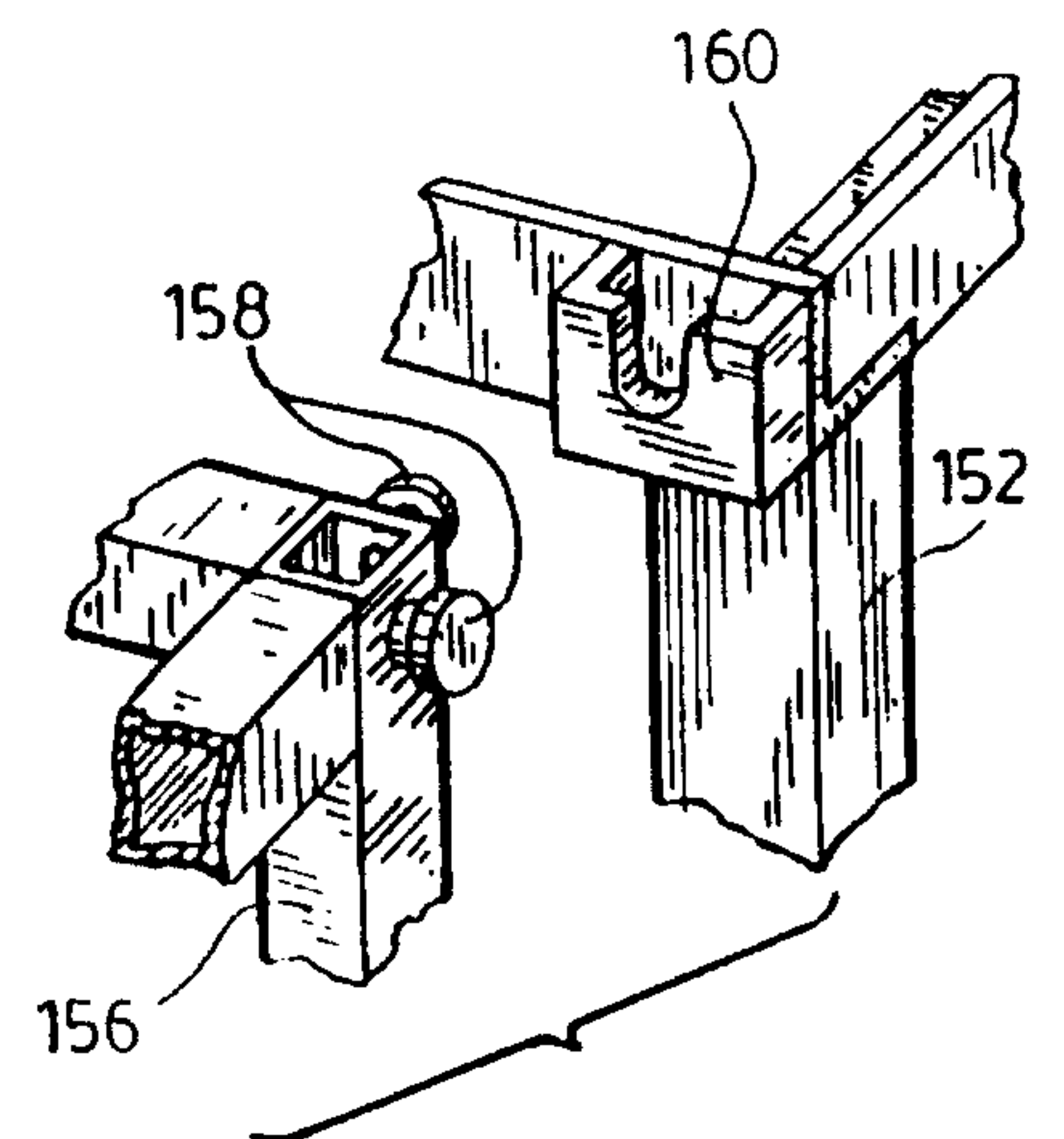


Fig.24

