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(54) **Title: SHAFT SEAL ASSEMBLY**

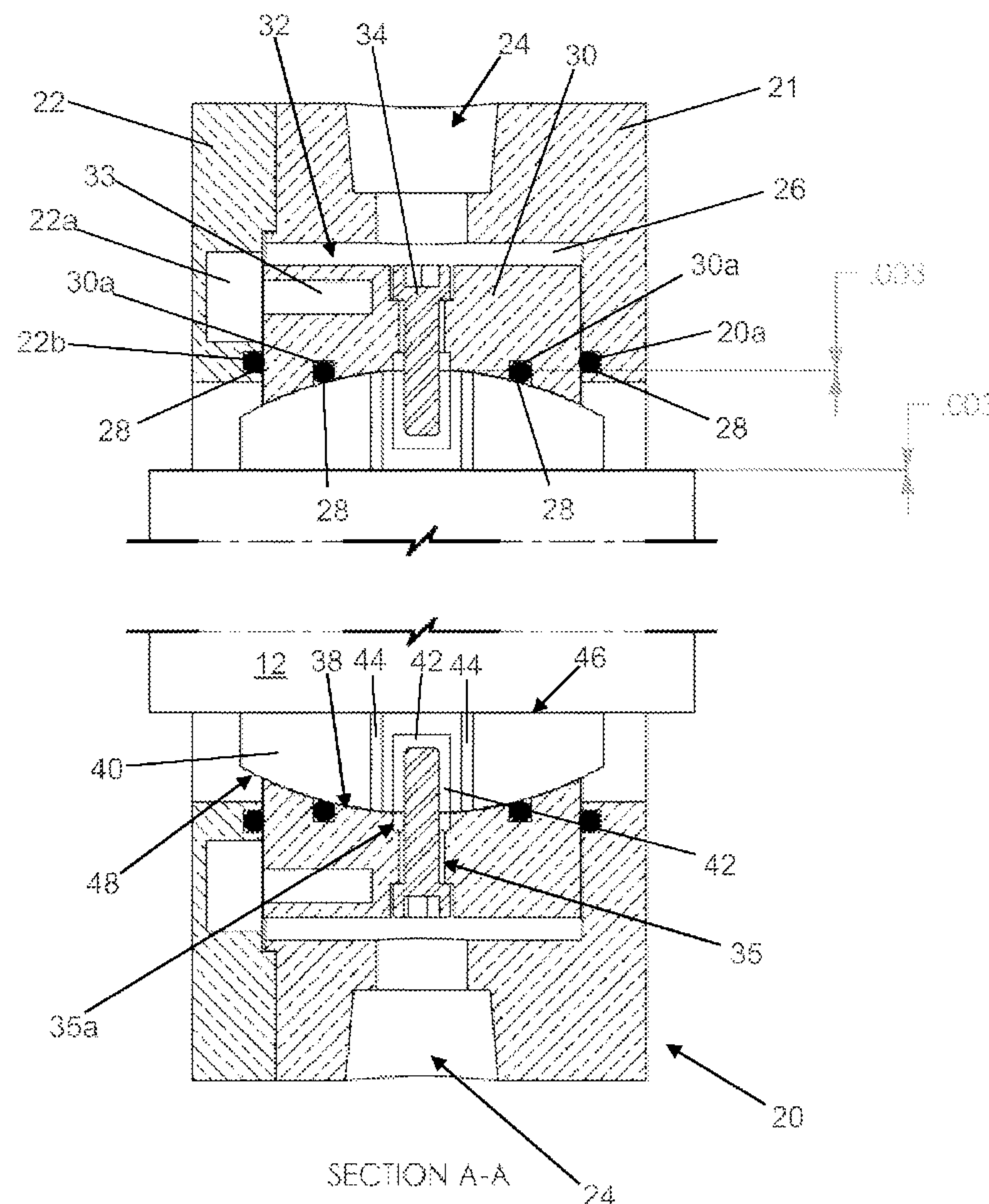


FIG. 16B

(57) Abrégé/Abstract:

An illustrative embodiment of a shaft seal assembly generally includes a fixed stator, a floating stator, and a sealing member. In the illustrative embodiment the fixed stator may be formed with one or more fluid conduits that are in fluid communication with one or

(57) Abrégé(suite)/Abstract(continued):

fluid conduits formed in the floating stator. The fluid conduits in the floating stator may be in fluid communication with one or more fluid conduits formed in a sealing member. A rotational interface may exist between the sealing member and a shaft, and the various fluid conduits may be configured to create a fluid barrier at that interface. Other embodiments of a shaft seal assembly may create a fluid barrier at a rotational interface between a rotor and a floating stator.

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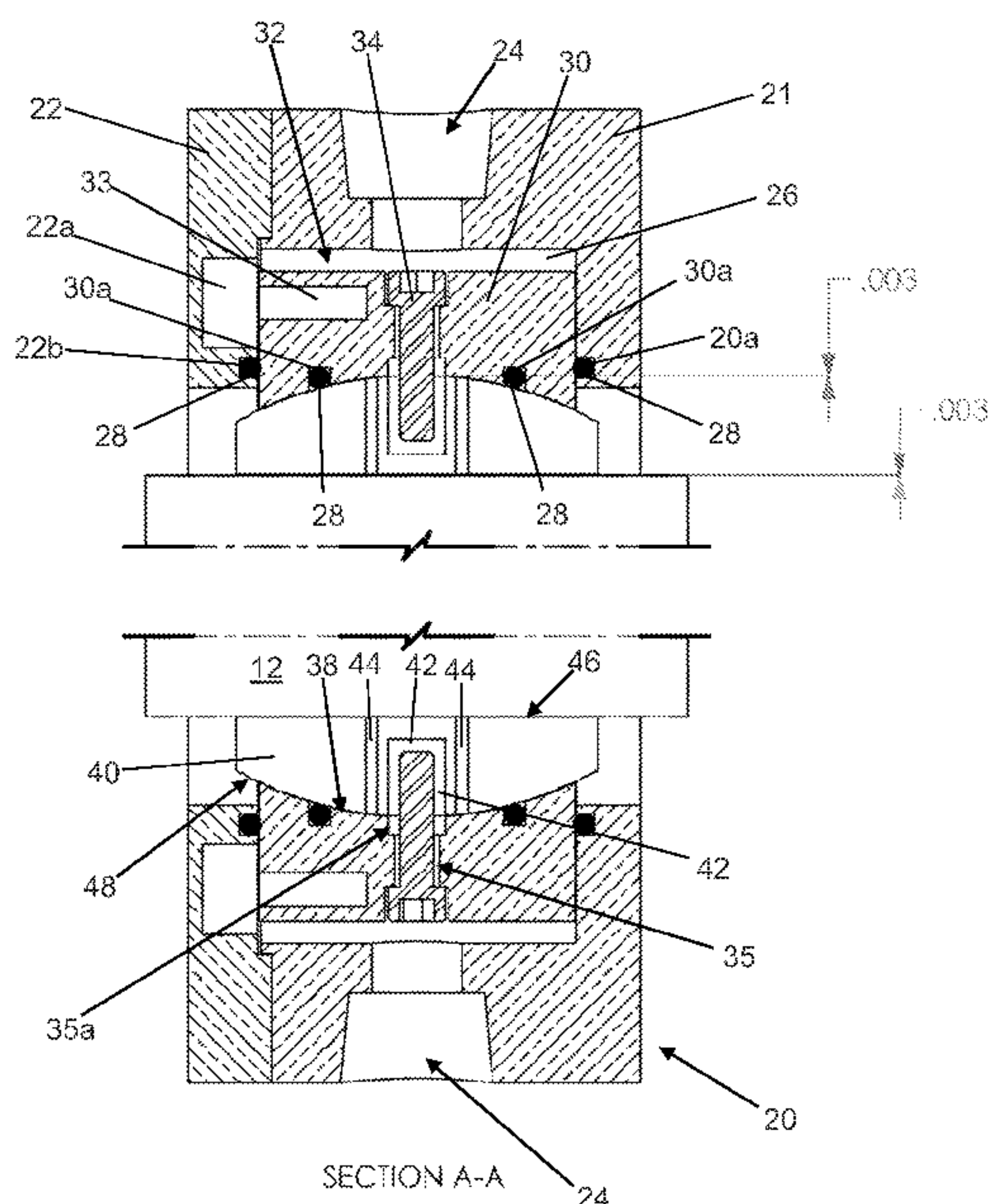


FIG. 16B

(57) Abstract: An illustrative embodiment of a shaft seal assembly generally includes a fixed stator, a floating stator, and a sealing member. In the illustrative embodiment the fixed stator may be formed with one or more fluid conduits that are in fluid communication with one or fluid conduits formed in the floating stator. The fluid conduits in the floating stator may be in fluid communication with one or more fluid conduits formed in a sealing member. A rotational interface may exist between the sealing member and a shaft, and the various fluid conduits may be configured to create a fluid barrier at that interface. Other embodiments of a shaft seal assembly may create a fluid barrier at a rotational interface between a rotor and a floating stator.

TITLE OF INVENTION

Shaft Seal Assembly

CROSS REFERENCE TO RELATED APPLICATIONS

This patent application claims priority from provisional U.S. Pat. App. No. 61/884,880 filed on 09/30/2013, this patent application also claims priority from non-provisional U.S. Pat. App. No. 14/500,003 filed on 09/29/2014, all of which are incorporated by reference herein in their entirety.

FIELD OF THE INVENTION

The present invention relates to a shaft seal assembly with multiple embodiments. In certain embodiments, the shaft seal assembly may be used as a product seal between a product vessel and a shaft therein.

STATEMENT REGARDING FEDERALLY SPONSORED RESEARCH OR DEVELOPMENT

No federal funds were used to create or develop the invention herein.

REFERENCE TO SEQUENCE LISTING, A TABLE, OR A COMPUTER PROGRAM LISTING COMPACT DISK APPENDIX

N/A

BRIEF DESCRIPTION OF THE DRAWINGS

In order that the advantages of the invention will be readily understood, a more particular description of the invention briefly described above will be rendered by reference to specific embodiments illustrated in the appended drawings. Understanding that these drawings depict only typical embodiments of the invention and are not therefore to be considered limited of its scope, the invention will be described and explained with additional specificity and detail through the use of the accompanying drawings.

FIG. 1 is a perspective exterior view of the shaft seal assembly.

FIG. 2 is an exterior end view of the shaft seal assembly with the shaft element aligned.

FIG. 3 is a sectional view of a first embodiment of the shaft seal assembly, as shown in FIG. 2 and mounted to a housing.

FIG. 3A illustrates the first surface seal-shaft integrity during angular and radial shaft alignment.

FIG. 3B illustrates second surface seal-shaft integrity during angular and radial shaft alignment.

FIG. 4 is an exterior end view with the shaft misaligned.

FIG. 5 is a sectional view of the first embodiment as shown in FIG. 3 with both angular and radial misalignment of the shaft applied.

FIG. 5A illustrates first seal-shaft integrity allowed by articulation during angular and radial shaft misalignment.

FIG. 5B illustrates second seal-shaft integrity allowed by articulation during angular and radial shaft misalignment.

FIG. 6 is a sectional view of a second embodiment of the shaft seal assembly as shown in FIG. 2.

FIG. 7 is a sectional view of a third embodiment as shown in FIG. 2.

FIG. 8 is a perspective view of a fourth embodiment as mounted to a vessel wall.

FIG. 9 is a cross-sectional view of a first embodiment of the pressure balanced shaft seal assembly mounted to a housing wherein the shaft is in alignment.

FIG. 9A is a detailed view of the portion of the first embodiment of the pressure balanced shaft seal assembly adjacent the vent wherein the shaft is in alignment.

FIG. 9B is a detailed view of the portion of first embodiment of the pressure balanced shaft seal assembly adjacent the fluid return pathway wherein the shaft is in alignment.

FIG. 10 is a cross-sectional view of the first embodiment of the pressure balanced shaft seal assembly shown during shaft misalignment.

FIG. 10A is a detailed view of the portion of the first embodiment of the pressure balanced shaft seal assembly adjacent the vent wherein the shaft is misaligned.

FIG. 10B is a detailed view of the portion of the first embodiment of the pressure balanced shaft seal assembly adjacent the fluid return pathway wherein the shaft is misaligned.

FIG. 11 is a cross-sectional view of a second embodiment of the pressure balanced shaft seal assembly wherein the shaft is in alignment.

FIG. 12 is a cross-sectional view of a third embodiment of the pressure balanced shaft seal assembly wherein the shaft is in alignment.

FIG. 13 is a cross-sectional view of another embodiment of a bearing isolator (or shaft seal assembly) configured with a rotor.

FIG. 14 is a cross-sectional view of a portion of yet another embodiment of a bearing isolator (or shaft seal assembly) configured with a rotor.

FIG. 14A is a cross-sectional view of the portion of the embodiment of a bearing isolator shown in FIG. 14 with the shaft misaligned and/or radially displaced.

FIG. 15 is a partial cross-sectional view of the embodiment of a bearing isolator shown in FIG. 13 wherein the shaft is misaligned and/or radially displaced.

FIG. 15A is a detailed view of a portion of the embodiment bearing isolator shown in FIG. 15.

FIG. 16A is an exterior face view of an illustrative embodiment of a multi-hole shaft seal assembly, wherein certain hidden surfaces are shown with broken lines.

FIG. 16B is a cross-sectional view of the embodiment of a multi-hole shaft seal assembly shown in FIG. 16A along line A-A.

FIG. 17A is an exterior face view of one embodiment of a sealing member that may be used with various embodiments of a multi-hole shaft seal assembly.

FIG. 17B is a cross-sectional view of the embodiment of a sealing member shown in FIG. 17A along line K-K.

DETAILED DESCRIPTION—ELEMENT LISTING (FIGS. 1–12)

Description	Element No.
Shaft	1
Fixed stator	2
Fixed stator (part-line)	2a
Labyrinth seal	3
Radiused face	3a
Floating stator	4
Fluid return pathway	5
Shaft seal clearance	6
First o-ring	7
Anti-rotation pin	8
Vent	9
Anti-rotation groove (floating stator)	10
Spherical interface	11
Anti-rotation pin	12
Second o-ring	13
Labyrinth seal pattern grooves	14
First o-ring channel	15
Cavity for anti-rotation device (fixed stator)	16
Axial face of labyrinth seal	17
Axial face of floating stator	18
Second o-ring channel	19
First clearance between floating stator/fixed stator	20
Second clearance between floating stator/fixed stator	21
Throttle groove	22
Labyrinth pattern annular groove	23
Sleeve	24
Shaft seal assembly	25
Throttle (alignment skate)	26
Floating stator annular groove	27
Labyrinth seal passage	28

Floating stator passage	29
Housing	30
Angle of misalignment	31
Bearings and bearing cavity	32
Mounting bolts	33
Vessel wall	34
Pressure balanced shaft seal assembly	40
Labyrinth seal interior face	42
Floating stator interior face	44
Pressure balancing annular channel	46
First radial interface	47a
Second radial interface	47b
Fixed stator annular groove	48
Annular groove radial-interior surface	48a

DETAILED DESCRIPTION

Before the various embodiments of the present invention are explained in detail, it is to be understood that the invention is not limited in its application to the details of construction and the arrangements of components set forth in the following description or illustrated in the drawings. The invention is capable of other embodiments and of being practiced or of being carried out in various ways. Also, it is to be understood that phraseology and terminology used herein with reference to device or element orientation (such as, for example, terms like "front", "back", "up", "down", "top", "bottom", and the like) are only used to simplify description of the present invention, and do not alone indicate or imply that the device or element referred to must have a particular orientation. In addition, terms such as "first", "second", and "third" are used herein and in the appended claims for purposes of description and are not intended to indicate or imply relative importance or significance. Furthermore, any dimensions recited or called out herein are for exemplary purposes only and are not meant to limit the scope of the present disclosure in any way unless so recited in the claims.

Figures 1–5 provide various views of a first illustrative embodiment of the shaft seal assembly 25 that allows for sealing various lubricating solutions within bearing housing 30 and/or preventing ingress of contaminants to the housing 30, which may be configured as a

bearing housing. Figures 6 and 7 provide alternative illustrative embodiments of the shaft seal assembly 25 wherein sealing fluids are used. Applicant herein defines sealing fluids to include at least both liquids and vapors. Applicant considers air, nitrogen, water and steam as well as any other fluid that may work with the proposed shaft seal assembly to provide a pressurized fluid barrier for any and all embodiments disclosed herein to be within the purview of the present disclosure. The gas or fluid chosen may be based at least upon process suitability with the product to be sealed.

Figure 1 is a perspective exterior view of the first illustrative embodiment of a shaft seal assembly 25 arranged and engaged with a shaft 1 inserted through the fixed stator 2 of shaft seal assembly 25. Figure 2 is an exterior end view of the shaft seal assembly with shaft 1 aligned within the shaft seal assembly 25.

Figure 3 is a sectional view of a first embodiment of the shaft seal assembly 25 shown in FIG. 2 illustrating that the shaft seal assembly 25 may be configured as a labyrinth seal for retaining lubrication solution within the bearing cavity 32 of housing 30 and/or preventing ingress of contaminants into the housing 30. The shaft 1 shown in FIG. 3 may experience radial, angular or axial movement relative to the fixed stator 2 or a portion thereof at various times. The fixed stator 2 of the shaft seal assembly 25 may be engaged with a housing 30 via any suitable method and/or structure, including but not limited to flange-mounted or press-fit. The shaft seal assembly 25 may also be used in applications with a rotating housing and stationary shaft. (Not shown) As required by the particular application of the shaft 1 and/or shaft seal assembly 25, the shaft 1 may be allowed to move freely in the axial direction in relation to the shaft seal assembly 25.

A labyrinth seal 3 having an interior surface may be positioned adjacent shaft 1. A defined clearance 6 may exist between the interior surface of said labyrinth seal 3 and the shaft 1. A radiused surface 3a may be configured such that it is opposite the interior surface of the labyrinth seal 3. The radiused surface 3a of the labyrinth seal 3 and the interior of the floating stator 4 may be configured to form a spherical interface 11. O-ring channels 15 and o-rings 7 may be disposed to cooperate with the radiused surface 3a of the labyrinth seal 3 to seal (or trap) fluid migration through, between and along engaged labyrinth seal 3 and floating stator 4 while maintaining a spherical interface 11, which spherical interface 11 may allow limited relative rotational movement (articulation) between labyrinth seal 3 and floating stator 4.

O-ring channels 15, as shown, may be machined into the floating stator 4 and may be positioned at the spherical interface 11 with labyrinth seal 3. O-ring channels 15 may be configured such that they are annular and continuous in relation to labyrinth seal 3. The o-ring channel 15 and o-ring 7 may also be placed in the labyrinth seal 3 adjacent the spherical interface 11. In certain embodiments, o-rings 7 may be constructed of materials that are compatible with both the product to be sealed and the preferred sealing fluid. O-ring channels 15 and o-rings 7 are but one possible combination of structures that may be used to seal various portions within the shaft seal assembly 25. Any other structures and/or method suitable for the particular embodiment of a shaft seal assembly 25 may be used without limitation.

Strategically placed anti-rotation pin(s) 12 may be inserted into anti-rotation grooves 10 and may serve to limit relative rotational movement between labyrinth seal 3 and floating stator 4. A plurality of anti-rotation grooves 10 and pins 12 may be placed around the radius of the shaft 1. If the shaft seal assembly 25 is used in combination with a sealing fluid, strategic anti-rotation pins 12 may be removed allowing corresponding anti-rotation grooves 10 to serve as a fluid passage through vent 9 and lubricant return 5, one illustrative embodiment of which is shown in FIG. 7. Additionally, the relationship of the diameters of anti-rotation pins 12 and anti-rotation grooves 10 may be selected to allow more or less angular misalignment of the shaft 1, respectively. For example, a relatively small-diameter anti-rotation pin 12 used with a large-diameter anti-rotation groove 10 would allow for greater relative movement of the labyrinth seal 3 in relation to the floating stator 4 in response to angular misalignment of shaft 1. A labyrinth seal 3 is one possible embodiment of a sealing structure that may be used adjacent to the shaft 1 within the shaft seal assembly 25. However, other structures and/or methods may be used to achieve similar functionality without limitation.

An annular channel may be formed within fixed stator 2 and may be defined by clearance 20 and 21 as allowed between the exterior of said floating stator 4 and the interior of the fixed stator 2 of shaft seal assembly 25. The annular channel of fixed stator 2 is highlighted as A-A' in FIG. 2. The annular channel of the fixed stator 2 may be formed with interior surfaces that are configured such that they are substantially perpendicular to said shaft 1. The exterior surfaces of the floating stator 4, which may be substantially encompassed within the annular channel of the fixed stator 2, may cooperatively engage with the first and second interior

perpendicular faces of the fixed stator 2. An inner interface may be formed by the first (shaft seal assembly 25 inboard side) perpendicular annular channel surface of the fixed stator 2 engaging with the first (inboard side) perpendicular face of the floating stator 4. An outer interface may be formed by the second (shaft seal assembly 25 outboard side) perpendicular annular interior channel surface of the fixed stator 2 engaging with the second (outboard side) perpendicular face of the floating stator 4. O-ring channels 19 and o-rings 13 may be disposed therein and may cooperate with the surfaces of floating stator 4 that are in perpendicular to relation to shaft 1. These o-rings 13 may function to seal (or trap) fluid migration between and along engaged floating stator 4 while allowing limited relative rotational movement between floating stator 4 and fixed stator 2. Floating stator 4 and fixed stator 2 are one possible embodiment of cooperatively engaged portions of a shaft seal assembly 25 that may be configured to allow relative motion between the portions in at least one dimension, and which may be used in combination with labyrinth seal 3 within the shaft seal assembly 25. However, other structures and/or methods may be used to achieve similar functionality without limitation.

O-ring channels 19 may be configured such that they are annular and continuous in relation to shaft 1. In an embodiment not shown herein, the o-ring channels 19 and o-rings 13 may be placed in the body of the floating stator 4 rather than the fixed stator 2. It is contemplated that for many applications it may be optimal to place those o-ring channels 19 and corresponding o-rings 13 in similar proximal relation. In certain embodiments, o-rings 7 may be constructed of materials that are compatible with both the product to be sealed and the preferred sealing fluid. O-ring channels 15 and o-rings 7 are but one possible combination of structures that may be used to seal various portions within the shaft seal assembly 25. Any other structures and/or method suitable for the particular embodiment of a shaft seal assembly 25 may be used without limitation.

Strategically placed anti-rotation pin(s) 8 may be inserted into anti-rotation groove(s) 16 and may serve to limit both relative radial and rotational movement between floating stator 4 and interior side of fixed stator 2. A plurality of anti-rotation grooves 16 and pins 8 may be placed around the radius of the shaft 1. The relationship of the diameters of anti-rotation pins 8 and anti-rotation grooves 16 may also be selected to allow more or less angular misalignment of the shaft. For example, a small-diameter anti-rotation pin 8 and large-diameter fixed stator

anti-rotation groove may allow for greater relative movement of the labyrinth seal 3 in response to angular misalignment of shaft 1.

The labyrinth pattern seal grooves 14 may be pressure equalized by venting through one or more vents 9. If so desired, the vents 9 may be supplied with a pressurized sealing fluid such that the sealing fluid over-pressurizes the labyrinth area 14 and shaft seal clearance 6 to increase the efficacy of shaft seal assembly 25. A spherical interface 11 between the labyrinth seal 3 and the floating stator 4 may be configured to allow for angular misalignment between the shaft 1 and fixed stator 2. O-ring channels 19 are annular with the shaft 1 and, as shown, may be machined into the fixed stator 2 and positioned at the interface between the fixed stator 2 and floating stator 4. O-ring channel 19 may also be placed in the floating stator 4 and may be engaged with o-rings 13, which may be configured to provide sealing contact with the fixed stator 2.

Figure 3A illustrates seal-shaft integrity during angular and radial shaft 1 alignment. This view highlights the alignment of the axial face 17 of the labyrinth seal 3 and the axial face 18 of the floating stator 4. Particular focus is drawn to the alignment of the axial faces 17, 18 at the spherical interface 11 between the floating stator 4 and labyrinth 3. Figure 3B illustrates the shaft-seal integrity during angular and radial shaft 1 alignment at the surface opposite that shown in Figure 3A. This view highlights the alignment of the axial faces 17, 18 of labyrinth seal 3 and floating stator 4, respectively, for the opposite portion of the shaft seal assembly 25 as shown in FIG. 3A. Those of ordinary skill in the art will appreciate that because the shaft 1 and the illustrative embodiments of a shaft seal assembly 25 are of a circular shape and nature, the surfaces are shown 360 degrees around shaft 1. Again, particular focus is drawn to the alignment of the axial faces 17, 18 at the spherical interface 11 between the labyrinth seal 3 and floating stator 4. Figures 3A and 3B also illustrate the first defined clearance 20 between the floating stator 4 and the fixed stator 2 and the second defined clearance 21 between the floating stator 4 and fixed stator 2 and opposite the first defined clearance 20.

In Figures 2, 3, 3A and 3B, the shaft 1 is not experiencing radial, angular or axial movement with respect to a housing 30. Accordingly, in the illustrative embodiments the width of the defined clearances 20 and 21, which may be substantially equal, may indicate little movement or misalignment upon the floating stator 4.

Figure 4 is an exterior end view of the shaft seal assembly 25 with the rotatable shaft 1 misaligned therein. Figure 5 is a sectional view of the first embodiment of the shaft seal assembly 25 as shown in FIG. 3 with both angular and radial misalignment of the shaft 1 applied. The shaft 1 as shown in FIG. 5 is also of the type that may experience radial, angular or axial movement relative to the fixed stator 2 (and/or housing 30) of the shaft seal assembly 25.

As shown at FIG. 5, the defined radial clearance 6 of labyrinth seal 3 with shaft 1 may be maintained even though the angle of shaft misalignment 31 has changed. The shaft 1 still may be allowed to move freely in the axial direction even though the angle of shaft misalignment 31 has changed. The arrangement of the shaft seal assembly 25 may allow the labyrinth seal 3 to move with the floating stator 4 upon introduction of radial movement of said shaft 1.

The labyrinth seal 3 and floating stator 4 may be secured together by one or more compressed o-rings 7 or any other suitable structure and/or method. Rotation of the labyrinth seal 3 within the floating stator 4 may be prevented by anti-rotation members, which may include but are not limited to screws, anti-rotation pins 8, or similar devices to inhibit rotation. The pins as shown in FIGS. 3, 3A, 3B, 5, 6 and 7 are one structure for preventing rotation of the labyrinth seal 3 and floating stator 4. However, any other suitable structure and/or method may be used to achieve similar results without limitation.

Lubricant, sealing fluid, or other media may be collected and drained through a series of one or more optional drains or lubricant return pathways 5. The labyrinth seal 3 may be pressure-equalized by venting through one or more vents 9. If so desired, the vents 9 may be supplied with pressurized air or other gas or fluid media to over-pressurize the labyrinth seal 3 to increase seal efficacy. The combination of close tolerances between the cooperatively engaged mechanical portions of the shaft seal assembly 25 and pressurized sealing fluid may inhibit both product and contaminant contact with the internals of the shaft seal assembly 25. The spherical interface 11 between the labyrinth seal 3 and the floating stator 4 may be configured to allow for angular misalignment between the shaft 1 and fixed stator 2. O-ring channel 19 and o-ring 13, which may be disposed therein, may cooperate with the opposing faces of the floating stator 4, which may be configured such that they are substantially in perpendicular relation to the rotational axis of the shaft 1. In this manner, the o-rings 13 may

cooperate with the floating stator 4 to seal (or trap) fluid migration between and along the floating stator 4 while allowing relative radial movement between stator 4 and fixed stator 2.

Figure 5A illustrates seal-shaft integrity allowed by the shaft seal assembly 25 during angular and radial shaft 1 misalignment. This view highlights the offset or articulation of the axial faces 17 of the labyrinth seal 3 may have in relation the axial faces 18 of the floating stator 4 for a first portion of the shaft seal assembly 25. Particular focus is drawn to the offset of the axial faces 17, 18 at the spherical interface 11 between labyrinth seal 3 and floating stator 4.

Figure 5B illustrates seal-shaft integrity for a second surface, opposite the first surface shown in Figure 5A, during angular and radial shaft misalignment. This view highlights that during misalignment of shaft 1, axial faces 17, 18, of the labyrinth seal 3 and floating stator 4, respectively, may not be aligned but instead move (articulate) in relation to each other. The shaft-to-seal clearance 6 may be maintained in response to the shaft 1 misalignment and the overall seal integrity may not be compromised because the seal integrity of the floating stator 4 to fixed stator 2 and the floating stator 4 to labyrinth seal 3 may be maintained during shaft 1 misalignment. Those of ordinary skill in the art will appreciate that because the shaft 1 and shaft seal assembly 25 may be circular in shape and nature, the surfaces are shown 360 degrees around shaft 1. Figures 5A and 5B also illustrate the first clearance or gap 20 between the floating stator 4 and the fixed stator 2 and the second clearance or gap 21 between the floating stator 4 and fixed stator 2 and opposite the first clearance or gap 20 during relative movement (other than rotational) between the shaft 1 and the housing 30.

In Figures 4, 5, 5A and 5B, the shaft 1 is experiencing radial, angular, or axial movement during rotation of the shaft 1 and the width of the gaps or clearances 20, 21 are shown as having changed in response to that movement as compared to the gaps or clearances 20, 21 depicted in FIGS. 3, 3A and 3B. The change in dimensions of clearance 20, 21 indicate the floating stator 4 may move in response to the movement or angular misalignment of shaft 1. The shaft seal assembly 25 may allow articulation between axial faces 17, 18, maintenance of spherical interface 11 and radial movement at first and second clearance, 20, 21, respectively, while maintaining shaft seal clearance 6.

Figure 6 is a sectional view of a second embodiment of the shaft seal assembly 25 as shown in FIG. 2 for over-pressurization with alternative labyrinth seal pattern grooves 14. In this

embodiment, the labyrinth seal pattern grooves 14 may be comprised of a friction-reducing substance such as polytetrafluoroethylene (PTFE), wherein the friction-reducing substance may be configured such that it forms a close clearance to the shaft 1. PTFE is also sometimes referred to as Teflon®, which is manufactured and marketed by Dupont. PTFE is a plastic with high chemical resistance, low and high temperature capability, resistance to weathering, low friction, electrical and thermal insulation, and high lubricity. Carbon or any other materials without limitation may be substituted for PTFE to provide the necessary sealing qualities and lubricous qualities for labyrinth seal pattern grooves 14.

Pressurized sealing fluids may be supplied to over-pressurize the lubricious labyrinth pattern 26 as shown in FIG. 6. The pressurized sealing fluids may be introduced to the annular groove 23 of the throttle 26 through one or more inlets. Throttle 26 may also be referred to as “an alignment skate” by those of ordinary skill in the art. Throttle 26 may allow the labyrinth seal 3 to respond to movement of the shaft 1 caused by the misalignment of the shaft 1. The pressurized sealing fluid may pass through the close clearance formed between the shaft 1 and labyrinth seal 3 having throttle 26. The close proximity of the throttle 26 to the shaft 1 also may create resistance to the sealing fluid flow over the shaft 1 and may cause pressure to build up inside the annular groove 23. Floating annular groove 27 in cooperation and connection with annular groove 23 also may provide an outlet for excess sealing fluid to be bled out of shaft seal assembly 25 for pressure equalization or to maintain a continuous fluid purge on the shaft sealing assembly 25 during operation. An advantage afforded by this aspect of the shaft sealing assembly 25 is its application wherein “clean-in place” product-seal decontamination procedures are preferred or required. Examples would include food grade applications.

Figure 7 illustrates shaft seal assembly 25 with the anti-rotation pin 12 removed to improve visualization of the inlets. These would typically be comprised of, but are not limited to, a series of ports, inlets or passages about the circumference of the shaft seal assembly 25. Figure 7 also illustrates that the shape and pattern of the labyrinth seal 3 may be varied from one embodiment of the shaft seal assembly 25 to the next. The shape of throttles 26 may also be varied as shown by the square profile shown at throttle groove 22 in addition to the circular-type 26. Also note that where direct contact with the shaft 1 is not desired, the shaft seal assembly 25 may be used in combination with a separate sleeve 24 that would be attached by varied means to the shaft 1.

Figure 8 shows that another embodiment of the shaft seal assembly 25 wherein the shaft seal assembly 25 has been affixed to a vessel wall 34. The shaft seal assembly 25 may be affixed to vessel wall 34 through securement members (e.g., including but not limited to mounting bolts 33) to ensure improved sealing wherein shaft 1 is subjected to angular misalignment. The mounting bolts 33 and slots (not numbered) through the shaft seal assembly 25 exterior are one structure and method of mounting the shaft seal assembly 25 to a housing 30. However, any suitable structure and/or method may be used without limitation.

In certain applications, especially those wherein the process side of shaft seal assembly 25 (generally the area to the left of the shaft seal assembly 25 as shown in FIGS. 3–3B and 5–7) is at an increased pressure, it is desirable for the shaft seal assembly 25 to be configured to balance the pressure experienced by the shaft seal assembly 25 in the axial direction. A pressure balanced shaft seal assembly 40 that balances the pressure (in the axial direction) that the product applies to the labyrinth seal interior face 42 and floating stator interior face 44 is shown in FIGS. 9–12.

In the first embodiment of the pressure balanced shaft seal assembly as shown in FIGS. 9–10B, the shaft sealing member (i.e., the labyrinth seal 3 in combination with the floating stator 4) includes a pressure balancing annular channel 46. Save for the pressure balancing annular channel 46, the pressure balanced shaft seal assembly 40 may operate in generally the same manner as the shaft seal assembly 25 shown in FIGS. 1–8 and described in detail above. That is, the floating stator 4 may be positioned in the fixed stator annular groove 48. The first clearance between floating stator/fixed stator 20, which in the embodiments pictured herein may be between the floating stator radial-exterior surface 45 and the annular groove radial-interior surface 48a (shown in FIGS. 9A and 9B), may account at least for radial perturbations of the shaft 1 with respect to the housing 30. The spherical interface 11 between the floating stator 4 and the labyrinth seal 3 may account at least for angular perturbations of the shaft 1 with respect to the housing 30.

The pressure balancing annular channel 46 may be formed in the floating stator 4 adjacent the first radial interface 47a between the floating stator 4 and the fixed stator 2, as shown in FIGS. 9–10 for the first embodiment. As shown in the various embodiments pictured herein, the first radial interface 47a between the floating stator 4 and the fixed stator 2 may be

adjacent the portion of the fixed stator 2 fashioned with the cavity for anti-rotation device 16. That is, the axial face of the floating stator 4 that is positioned within the fixed stator 2 and furthest from the process side of the pressure balanced shaft seal assembly 40. A second radial interface 47b between the floating stator 4 and fixed stator 2, which may be substantially parallel to the first radial interface 47a, may be positioned closer to the process side of the pressure balanced shaft seal assembly 40 as compared to the first radial interface 47a.

In many applications the optimal radial dimension of the pressure balancing annular channel 46 may be substantially similar to the radial dimension of the floating stator interior face 44 so that the area of the floating stator 4 acted upon by the product and the area of the floating stator 4 acted upon by the sealing fluid may have relatively equal surface areas. In such a configuration, the axial forces may generally balance if the product and the sealing fluid are pressurized to approximately the same value. Accordingly, the optimal radial dimension of the pressure balancing annular channel 46 may depend on the design characteristics of the entire system, and the radial dimension of the pressure balancing annular channel 46 may be any suitable amount for a particular application, whether greater or less than the radial dimension of the floating stator interior face 44. The axial dimension of the pressure balancing annular channel 46 may also vary depending on the design characteristics of the entire system, including but not limited to the specific sealing fluid that is used, the product pressure, and the pressure of the sealing fluid. In some applications the optimal axial dimension of the pressure balancing annular channel 46 will be 0.005 of an inch, but may be greater in other embodiments and less in still other embodiments.

The pressure balancing annular channel 46 may allow sealing fluid introduced into the first clearance between floating stator/fixed stator 20 (from where the sealing fluid may enter the pressure balancing annular channel 46) to act upon the floating stator 4 in an axial direction. Typically, the process side of the pressure balanced shaft seal assembly 40 (generally the area to the left of the pressure balanced shaft seal assembly 40 as shown in FIGS. 9–12) experiences forces from the process fluid acting upon the labyrinth seal interior face 42 and floating stator interior face 44. These forces are most often due to the pressure generated by the rotating equipment to which the shaft 1 is coupled. For example, if the shaft 1 is coupled to a fluid pump generating seventy pounds per square inch (psi) of head pressure, the process side of the pressure balanced shaft seal assembly 40 may be pressurized to approximately 70

psi. This pressurized fluid may act upon the labyrinth seal interior face 42 and floating stator interior face 44, and consequently urge the labyrinth seal 3 and floating stator 4 in the axial direction away from the process side of the pressure balancing shaft seal assembly 40 (i.e., generally to the right side of the drawing as depicted in FIGS. 9–12). By contrast, sealing fluid located in the pressure balancing annular channel 46 may urge the labyrinth seal 3 and floating stator 4 in the axial direction toward the process side of the pressure balancing shaft seal assembly 40, which may substantially cancel the axial force the product exerts upon the pressure balancing shaft seal assembly 40, depending on the design of the sealing fluid system.

Figures 11 and 12 show a second and third embodiment of the pressure balanced shaft seal assembly 40, respectively. The second and third embodiments of the pressure balanced shaft seal assembly 40 generally correspond to the second and third embodiments of the shaft seal assembly 25 as shown in FIGS. 7 and 8 and described in detail above. However, as with the first embodiment of the pressure balanced shaft seal assembly 40 as shown in FIGS. 9–10B, the second and third embodiments include a pressure balancing annular channel 46.

The various embodiments of the pressure balanced shaft seal assembly 40 pictured and described herein may be formed with a fixed stator 2 and floating stator 4 that may be comprised of two distinct portions. These embodiments may facilitate assembly of the pressure balanced shaft seal assembly 40 since in the embodiments pictured herein the majority of the floating stator 4 may be positioned within the fixed stator 2. When installing a pressure balanced shaft seal assembly 40 according to the first embodiment (as pictured in FIGS. 9–10B), the first portion of fixed stator 2 (i.e., the portion adjacent the process side of the pressure balanced shaft seal assembly 40) may be affixed to a housing 30. Next, the floating stator 4 and labyrinth seal 3 may be positioned as a singular assembled piece (wherein the components forming the spherical interface 11 have been preassembled) between the shaft 1 and the first portion of the fixed stator 2. The placement of the floating stator 4 and labyrinth seal 3 within the fixed stator 3 may form the second axial interface 47b between the fixed stator 2 and floating stator 4. Finally, the second portion of the fixed stator 2 (i.e., the portion furthest from the process side of the pressure balanced shaft seal assembly 40) may be positioned adjacent to and affixed to the first portion of the fixed stator 2. The positioning of the second portion of the fixed stator 2 subsequently may form the first radial interface 47a between the fixed stator 2 and floating stator 4.

Alternatively, the floating stator 4 and labyrinth seal 3 may be separately positioned within the fixed stator annular groove 48. For example, after the first portion of the fixed stator 2 has been affixed to the housing 30, the first portion of the floating stator 4 may be positioned within the fixed stator annular groove 48. The placement of the first portion of the floating stator 4 within the fixed stator annular groove 48 may form the second axial interface 47b between the fixed stator 2 and floating stator 4. Next, the labyrinth seal 3 may be positioned adjacent the shaft 3, the placement of which may form a portion of the spherical interface 11 between the floating stator 4 and labyrinth seal 3. Next, the second portion of the floating stator 4 may be positioned adjacent the first portion of the floating stator 4 and affixed thereto with a plurality of anti-rotation pins 8, which may complete the spherical interface 11 between the floating stator 4 and labyrinth seal 3. Finally, the second portion of the fixed stator 2 may be affixed to the first portion of the fixed stator 2 with a plurality of bolts, rivets, or other fasteners without limitation, the placement of which may form the first axial interface 47a between the floating stator 4 and fixed stator 2. Any suitable securing members known to those skilled in the art may be used to affix the first and second portions of the floating stator 4 to one another or to affix the first and second portions of the fixed stator 2 to one another in any embodiments of a shaft seal assembly 25 or pressure balanced shaft seal assembly 40 without limitation.

ELEMENT LISTING (FIGS. 13–15A)

Description	Element No.
Shaft	10
Bearing isolator	18
Housing	19
Rotor	20
Stator	30, 31a
Fixed stator	31
Passage	40, 40a
Spherical surface	50, 51
Clearance	52
Frictional seal	60

Flange unit	61a
Center point	80
Conduit	99
Fluid	100
Pin	101
Annular recess	102

Figure 13 shows another embodiment of a bearing isolator 18 (or shaft seal assembly) mounted adjacent a shaft 10. The shaft 10 may extend through the bearing isolator 18 and/or the housing 19. A source of gas or fluid, 100 which may include but is not limited water, gas, vapor and/or lubricant, may also be in communication with the bearing isolator 18 via conduit 99. The rotor 20 may be affixed to the shaft 10 by means by a frictional seal 60, which may be configured as one or more o-rings. The rotor 20 may be configured to follow the rotational movement of the shaft 10 because of the frictional engagement of the seals 60. The passages 40 and 40a may be configured as shown but will not be described in detail here because such description is already understood by those of ordinary skill in the art.

A pair of corresponding spherical surfaces 50 and 51 may be used to create a self-aligning radial clearance 52 between the rotor 20 and the stator 30 prior to, during, and after use. This clearance 52 may be maintained at a constant value even as the shaft 10 becomes misaligned during use. Various amounts and direction of misalignment between the centerline of the shaft 10 and the housing 19 are illustrated in FIGS. 15–17. An annular recess 102 between the stator 30 and fixed stator 31 may allow the bearing isolator 18 to accommodate a predetermined amount of radial shaft displacement.

In the embodiments shown herein, the spherical surfaces 50, 51 may have a center point identical from the axial faces of both the rotor and stator 20, 30, respectively. However, the spherical surfaces 50, 51 may be radially, and/or as shown, vertically spaced apart. These spherical surfaces 50, 51 may move radially in response to and/or in connection with and/or in concert with the radially positioning of other components of the bearing isolator 18. Typically, if the shaft 10 becomes misaligned with respect to the housing 19, the rotor 20

may consequently become misaligned with respect thereto, and then the spherical surfaces 50, 51 and/or the stator 30, moving radially within the annular recess of the fixed stator 31, may compensate for the misalignment.

Figures 15 and 15A illustrate that in one embodiment of the bearing isolator 18, the rotor 20 may move with respect to the stator 30, 31 as shaft 10 is misaligned with respect to housing 19 through the interaction between spherical surfaces 50, 51. Such relative movement to help to ensure the distances between the center points of the rotor 20 and stator 30 and a fixed point on the housing 19 are constant or relatively constant during use.

In the embodiment of the bearing isolator 18 shown in FIGS. 14 and 14A, the spherical surfaces 50, 51 may be positioned on a fixed stator 31 and stator 31a, respectively, rather than on the rotor 20 and stator 30. Still referring to FIGS. 14 and 14A, this design may allow the rotor 20 and stator 31a to move with respect to the fixed stator 31, flange unit 61a, and/or housing 19. The rotor 20, stator 31a, and fixed stator 31 may move radially with respect to the flange unit 61a (and consequently with respect to the housing 19) as best shown in FIG. 14A. In this embodiment of the bearing isolator 18 there may be a very minimal amount of relative rotation between the spherical surfaces 50, 51.

The embodiment of the bearing isolator 18 shown in FIGS. 14 and 14A may provide for controlled radial movement of the fixed stator 31, stator 31a, and/or rotor 20 with respect to flange unit 61a, which flange unit 61a may be engaged with a housing 19. Rotational movement of the fixed stator 30 with respect to the flange unit 61a may be prevented by anti-rotational pins 101. The fixed stator 31 may be frictionally secured to the flange unit 61a using a frictional seal 61, which may be made of any material with sufficient elasticity and frictional characteristics to hold the fixed stator 31 in a fixed radial position with respect to the flange unit 61a but still be responsive to the radial forces when the shaft 10 is misaligned. Changes to the radial position of the fixed stator 31, stator 31a, and/or rotor 20 and the resulting positions thereof (as well as the resulting position of the interface between the fixed stator 31 and stator 31a) may occur until the radial force is fully accommodated or until the maximum radial displacement of the bearing isolator 18 is reached.

Referring now to FIGS. 15 and 15A, in operation the rotor 20 may be moved radially as the shaft 10 becomes misaligned with respect to the housing 19. Radial movement of the

spherical surfaces 50, 51 between the stator 31a and fixed stator 31 may result from this pressure. FIG. 15 shows potential resultant radial movement of center point 80 as the shaft 10 is misaligned. During normal operation, the shaft 10 is typically horizontal with respect to the orientation shown in FIG. 15, as represented by line A. As the shaft 10 becomes misaligned in a manner represented by line B, the center point 80 may move to a point along line A''. As the shaft 10 becomes misaligned in a manner represented by line B', the center point 80 may move to a point along line A'. However, in other shaft 10 misalignments, the radial positions of the rotor 20, stator 30, and/or fixed stator 31 may be constant and the spherical surfaces 50, 51 may compensate for the shaft 10 misalignment. From the preceding description it will be apparent that the bearing isolator 18 may provide a constant seal around the shaft 10 because the distance between the spherical surfaces 50, 51 may be maintained as a constant regardless of shaft 10 misalignment of a normal or design nature.

The physical dimensions of the spherical surfaces 50 and 51 may vary in linear value and in distance from the center point 80, depending on the specific application of the bearing isolator 18. These variations will be utilized to accommodate different sizes of shafts and seals and different amounts of misalignment, and therefore in no way limit the scope of the bearing isolator 18 as disclosed herein. Additionally, and suitable structure and/or method for engaging various elements with one another either rotationally, fixedly, or with various degrees of freedom of motion therebetween may be used with the shaft seal assembly 18 without limitation, including but not limited to screws, bolts, pins, chemical adhesives, interference fits, and/or combinations thereof.

ELEMENT LISTING (FIGS. 16–17)

Description	Element No.
Shaft seal assembly	10
Shaft	12
Fastener	14
Fixed stator	20
Fixed stator seal groove	20a
Main body	21
Face plate	22

Face plate pin recess	22a
Face plate seal groove	22b
Inlet	24
Annular recess	26
Seal	28
Floating stator	30
Floating stator seal groove	30a
Radial exterior surface	32
First pin recess	33
Pin	34
Second pin recess	35
Second pin recess enlarged portion	35a
Floating stator annular groove	37
Concave surface	38
Sealing member	40
Recess	42
Radial bore	44
Radial bore inlet	44a
Radial bore outlet	44b
Radial interior surface	46
Convex surface	48

Another embodiment of a shaft seal assembly 10 is shown in FIGS. 16A & 16B. This embodiment is similar to the embodiment of the shaft seal assembly 25 described above and shown in FIGS. 1–12. The shaft seal assembly 10 may include a fixed stator 20, floating stator 30, and a sealing member 40, as shown. In the pictured embodiment, the sealing member 40 may be positioned adjacent a shaft 12 that is rotatable with respect to the shaft seal assembly 10 and/or housing. Accordingly, a rotational interface may exist between a radial interior surface 46 of the sealing member 40 positioned adjacent the shaft 12 and an exterior portion of the shaft 12. In other embodiments of the shaft seal assembly 10 not pictured herein, the sealing member 40 may be engaged with the shaft 12 such that it rotates

therewith (e.g., the shaft seal assembly 10 may be configured with a rotor). In such an embodiment, a rotational interface may exist between a concave surface 38 of the floating stator 30 and a convex surface 48 of the sealing member 40. Accordingly, the scope of the shaft seal assembly 10 as disclosed herein extends to shaft seal assemblies 10 in which the sealing member 40 does or does not rotate with a shaft 12.

The embodiment of the shaft seal assembly 10 shown in FIGS. 16A & 16B may include a fixed stator 20 that may be securely mounted to a housing (not shown in FIGS. 16A & 16B) by any suitable methods and/or structure. The fixed stator 20 may include a main body 21 and a face plate 22 that may be engaged with one another via one or more fasteners 14. It is contemplated that a fixed stator 20 formed with a main body 21 and face plate 22 may facilitate ease of installation of the shaft seal assembly 10 in certain applications. In such applications, the main body 21 may be affixed to the housing, the sealing member 40 and floating stator 30 may be positioned appropriately, and then the face plate 22 may be secured to the main body 21. However, the scope of the present disclosure is in no way limited by the specific mounting and/or installation method of the shaft seal assembly 10.

The fixed stator 20 may be formed with an annular recess 26 into which a portion of the floating stator 30 and/or sealing member 40 may be positioned. A predetermined clearance between the radial exterior surface 32 of the floating stator 30 (as well as the axial exterior surfaces thereof) and the interior surfaces of the annular recess 26 may be selected to allow for a predetermined amount of relative radial and/or axial movement between the fixed stator 20 and floating stator 30. At least one pin 34 (which may be radially oriented as in the embodiment shown in FIGS. 16A & 16B) may be engaged with the floating stator 30 at a second pin recess 35, and a portion of the pin 34 may extend into a recess 42 formed in the sealing member 40. Additionally, other pins (not shown, but which may be axially oriented) also may be engaged with the floating stator about a first pin recess 33, and a portion of that pin may extend into a face plate pin recess 22a. In the illustrative embodiment shown in FIGS. 16A & 16B, the pins 35 may mitigate relative rotation between the floating stator 30 and the sealing member 40. Axially oriented pins (not shown) may mitigate relative rotation between the floating stator 30 and the fixed stator 20. The axial interfaces between the floating stator 30 and fixed stator 20 may be sealed with seals 28, which seals 28 may be positioned in fixed stator seal grooves 20a and/or face plate seal grooves 22b. The seals 28

may be configured as o-rings, but may be differently configured in other embodiments of the shaft seal assembly 10 without limitation.

The floating stator 30 may also be formed with a concave surface 38 in a radial interior portion thereof. This concave surface 38 may form a semi-spherical interface with a corresponding convex surface 48 formed in the radial exterior portion of the sealing member 40. Accordingly, the shaft seal assembly 10 shown in FIGS. 16A & 16B may accommodate shaft 12 misalignment, shaft 12 radial movement, and shaft 12 axial movement with respect to the shaft seal assembly 10 and/or equipment housing in an identical and/or similar manner to that previously described for the shaft seal assemblies 25 shown in FIGS. 1–12.

The illustrative embodiment of the shaft seal assembly 10 also may include various fluid conduits for applying a sealing fluid to the shaft seal assembly 10. The fixed stator 20 may be formed with one or more inlets 24 for introduction of a sealing fluid to the shaft seal assembly 10. The inlet 24 may be in fluid communication with the annular recess 26 formed in the fixed stator 20, which annular recess 26 may be in fluid communication with one or more radial passages (not shown) formed in the floating stator 30 and extending from the radial exterior surface 32 thereof to the concave surface 38 thereof. Alternatively, or in addition to the one or more radial passages, the second pin recess 35 formed in the floating stator 30 may be configured to allow a specific amount of sealing fluid to traverse the length of the second pin recess 35 in a radially inward direction. The radially interior terminus of the second pin recess 35 may be formed with a second pin recess enlarged portion 35a.

Alternatively, the floating stator 30 may be formed with a floating stator annular groove 37 on the concave surface 38 thereof. These radial passages, second pin recess 35, second pin recess enlarged portion 35a, and/or floating stator annular groove 37 may serve as a conduit for sealing fluid from the annular recess 26 of the fixed stator 20 to the convex surface 48 of the sealing member 40. Accordingly, the scope of the shaft seal assembly 10 is not limited by the specific combinations of fluid conduits disclosed herein, but extends to all configurations of fluid conduits that may supply a sealing fluid to the sealing member 40.

The fixed stator 20 and/or seals 28 between the fixed stator 20 and floating stator 30 may be configured so that the majority of sealing fluid introduced to the inlet 24 passes through the floating stator 30 (by any fluid conduit configuration, as explained above) in a radially inward direction. The semi-spherical interface between the floating stator 30 concave surface

38 and the sealing member 40 convex surface 48 may be sealed with seals 28, which seals 28 may be positioned in floating stator seal grooves 30a and/or sealing member seal grooves (not shown). The seals 28 may be configured as o-rings, but any suitable structure and/or method may be used without limitation. The floating stator 30, sealing member 40, and/or seals 28 therebetween may be configured so that the majority of sealing fluid exiting the floating stator 30 passes through the sealing member 40 through a plurality of radial bores 44 in a direction from the convex surface 48 of the sealing member 40 to the radial interior surface 46 thereof (i.e., in a generally radially inward direction, such that the sealing fluid exits the shaft seal assembly 10 adjacent the shaft 12).

The fixed stator 20, floating stator 30, and/or sealing member 40 may be configured such that the fluid conduits formed therein allow the majority of sealing fluid to exit the shaft seal assembly 10 from an area between the sealing member 40 and shaft 12 at a predetermined rate for a given set of operation parameters (e.g., sealing fluid viscosity, pressure, and/or volumetric flow rate, rpm of shaft 12, etc.). The illustrative embodiment of the shaft seal assembly 10 may be formed with thirty two (32) radial bores 44 in the sealing member 40 in corresponding pairs equally spaced about the circumference of the sealing member, which is best shown in FIGS. 17A & 17B. Each radial bore 44 may be formed with a radial bore inlet 44a adjacent the convex surface 48 and a radial bore outlet 44b adjacent the radial interior surface 46. However, in other embodiments of the sealing member 40 not shown herein, the sealing member 40 may be configured with differently configured radial bores 44, different numbers of radial bores 44, and/or different relative positions of radial bores 44 without limitation.

It is contemplated that the configuration of radial bores 44 shown in the embodiment of a sealing member 40 pictured in FIGS. 17A and 17B may be more efficient than other configurations in that a lower volumetric flow rate of sealing fluid may be required for a given set of operational parameters when compared to the prior art. Additionally, the smooth, generally cylindrical configuration of the radial interior surface 46 may create a pressurized fluid barrier between the shaft 12 and the sealing member 40 at the interface thereof (e.g., a “lift-off” seal). This may lead to a nearly frictionless shaft seal assembly 10 with no and/or minimal contact between the shaft 12 and the sealing member 40 during operation. However, in other embodiments, different numbers, spacing, and/or configurations of the fluid conduits in the fixed stator 20, floating stator 30, and/or sealing member 40 may be used without

departing from the spirit and scope of the shaft seal assembly 10 as disclosed and claimed herein.

In light of the present disclosure, it will be apparent to those skilled in the art that the configuration of fluid conduits disclosed herein may be adapted to create a pressurized fluid barrier between any interface at which two elements are rotating with respect to one another, such as the articulated seal disclosed in U.S. Pat. No. 7,090,403. U.S. Pat. No. 7,090,403 is incorporated by reference herein in its entirety, and discloses embodiments of a shaft seal assembly having a spherical rotational interface between a rotor and a floating stator (such as those shown in FIGS. 13, 15, and 15A) and embodiments of a shaft seal assembly having a generally non-rotating spherical interface between two portions of a stator (such as those shown in FIGS. 14 and 14A). Accordingly, the scope of the shaft seal assembly 10 as disclosed herein is not limited by the location and/or type of rotational interface the shaft seal assembly 10 is configured to accommodate.

For example, in an embodiment not pictured herein, the stator 30 of an embodiment similar to that shown in FIGS. 13, 15, and 15A may be configured with one or more generally narrow diameter radial bores (which may be generally similar to those shown in the embodiment in FIGS. 17A and 17B). Those radial bores may be configured so as to provide fluid from an external source (which may be in fluid communication with passage 40) to the interface between spherical surfaces on the stator portions 31, 31a (which may be configured as a concave surface on stator 31 and a convex surface on stator 31a). Alternatively, the stator 31 may be configured with radial bores that serve to provide fluid from an external source (which may be in fluid communication with passage 40) to the interface between stator 31a and rotor 20, which may be a rotational interface having a labyrinth seal pattern and/or one or more seals (which may be configured as o-rings) therein.

The specific configuration and/or physical dimensions of the various features of the fixed stator 20, floating stator 30, and/or sealing member 40 (e.g., the radial dimension of the annular recess 26, the surface area of the concave surface 38 and/or convex surface 48, the diameter, length, and orientation of the radial bores 44, etc.) may vary depending on the specific application of the shaft seal assembly 10. These variations may be utilized to accommodate different sizes of shafts 12 and/or shaft seal assemblies 10 and different amounts and/or types of relative movement between a shaft 12 and shaft seal assembly 10.

The materials used to construct the shaft seal assemblies 10, 25 and various elements thereof will vary depending on the specific application, but it is contemplated that bronze, brass, stainless steel, or other non-sparking metals and/or metallic alloys and/or combinations thereof will be especially useful for some applications. Accordingly, the above-referenced elements may be constructed of any material known to those skilled in the art or later developed, which material is appropriate for the specific application of the shaft seal assembly 10, 25, without departing from the spirit and scope of the shaft seal assemblies 10, 25 as disclosed and claimed herein.

Having described the preferred embodiments, other features of the shaft seal assemblies 10, 25 will undoubtedly occur to those of ordinary skill in the art, as will numerous modifications and alterations in the embodiments as illustrated herein, all of which may be achieved without departing from the spirit and scope of the shaft seal assemblies 10, 25 disclosed herein. Accordingly, the methods and embodiments pictured and described herein are for illustrative purposes only.

It should be noted that the shaft seal assemblies 10, 25 are not limited to the specific embodiments pictured and described herein, but are intended to apply to all similar apparatuses and methods for accommodating shaft(s) misalignment with respect to a housing and/or shaft seal assembly 10, 25, whether the misalignment is angular, radial, and/or axial; and for configuring a shaft seal assembly 10 to create a pressurized fluid barrier between a rotating element and a non-rotating element. Modifications and alterations from the described embodiments will occur to those skilled in the art without departure from the spirit and scope of the shaft seal assemblies 10, 25.

CLAIMS

What is claimed is:

1. A shaft seal assembly comprising:
 - a. a fixed stator configured for engagement with a housing, wherein said fixed stator includes an annular recess along a portion of the radial-interior surface thereof;
 - b. a shaft sealing member configured to be positioned in said annular recess, wherein said shaft sealing member comprises:
 - i. a floating stator, wherein a radial-exterior surface of said floating stator is separated from said annular groove radial-interior surface in the radial dimension by a predetermined amount to create a first clearance between said floating stator and said fixed stator, and wherein a radial-interior surface of said floating stator is substantially concave in shape; and,
 - ii. a sealing member, wherein said sealing member includes a radial-exterior surface with a substantially convex shape that corresponds to the radial-interior surface of said floating stator so as to create a spherical interface therebetween, and wherein said sealing member is configured to be positioned adjacent a shaft extend from said housing; and,
 - iii. a plurality of radial bores extending from said radial-exterior surface of said sealing member to said radial-interior surface thereof, and wherein a pressurized sealing fluid may be in fluid communication with said plurality of radial bores such that said pressurized sealing fluid moves through said plurality of radial bores in a radially inward direction.
2. The shaft seal assembly according to claim 1 wherein said plurality of radial bores is further defined as 16 radial bores evenly spaced about the periphery of said sealing member.
3. The shaft seal assembly according to claim 1 wherein said plurality of radial bores is further defined as being arranged in corresponding pairs, and wherein said plurality of radial bores are evenly spaced about the periphery of said sealing member.
4. The shaft seal assembly according to claim 1 wherein said plurality of radial bores is further defined as being unevenly spaced about the periphery of said sealing member.
5. The shaft seal assembly according to claim 1 wherein said fixed stator further comprises an inlet, and wherein said pressurized sealing fluid is provided to said inlet.

6. The shaft seal assembly according to claim 5 wherein said floating stator further comprises a fluid conduit, wherein said fluid conduit is in fluid communication with both said inlet and said spherical interface.
7. The shaft seal assembly according to claim 6 wherein said fluid conduit in said floating stator is further defined as being a second pin recess.
8. The shaft seal assembly according to claim 7 wherein said second pin recess further comprises a second pin recess enlarged portion on a radially inward portion thereof.
9. A shaft seal assembly comprising:
 - a. a stator having a spherical face on a portion thereof;
 - b. a rotor having a spherical face corresponding to said spherical face of said stator to create a semi-spherical interface between said rotor and said stator;
 - c. a plurality of radial bores formed in said stator, wherein said plurality of radial bores extend to said semi-spherical interface; and,
 - d. a passage formed in said stator, wherein said passage is configured to allow a pressurized sealing fluid to be in fluid communication with said plurality of radial bores.
10. The shaft seal assembly according to claim 9 wherein said plurality of radial bores is further defined as 16 radial bores evenly spaced about said semi-spherical interface.
11. The shaft seal assembly according to claim 9 wherein said plurality of radial bores is further defined as being arranged in corresponding pairs, and wherein said plurality of radial bores are evenly spaced about said semi-spherical interface.
12. The shaft seal assembly according to claim 9 wherein said plurality of radial bores is further defined as being unevenly spaced about said semi-spherical interface.
13. The shaft seal assembly according to claim 9 wherein said stator further comprises an inlet, wherein said pressurized sealing fluid is provided to said inlet, and wherein said inlet is in fluid communication with said passage.
14. The shaft seal assembly according to claim 9 wherein said stator is further defined as being a floating stator, wherein said shaft seal assembly further comprises a fixed stator, and wherein said floating stator is disposed in an annular recess formed in said fixed stator.
15. A shaft seal assembly comprising:
 - a. a first stator configured for engagement with a housing, wherein said first stator is formed with a radial-interior surface that is substantially concave in shape;

- b. a second stator positioned adjacent said first stator, wherein said second stator is formed with a radial-exterior surface that is substantially convex in shape so as to create a spherical interface between said first and second stators;
 - c. a rotor positioned adjacent said second stator, wherein said second stator and said rotor are configured such that a labyrinth pattern groove is positioned therebetween;
 - d. a plurality of radial bores extending from said radial-exterior surface of said second stator to said labyrinth pattern groove, wherein a pressurized sealing fluid may be in fluid communication with said plurality of radial bores such that said pressurized sealing fluid moves through said plurality of radial bores in a radially inward direction.
16. The shaft seal assembly according to claim 15 wherein said first stator further comprises an inlet, and wherein said pressurized sealing fluid may be introduced to said shaft seal assembly via said inlet.
17. The shaft seal assembly according to claim 16 wherein said second stator further comprises a fluid conduit, wherein said fluid conduit is in fluid communication with both said inlet and said plurality of radial bores.
18. The shaft seal assembly according to claim 17 further comprising a flange unit, wherein said flange unit is configured for engagement with a housing, wherein said first stator further comprises an annular recess on a radial-exterior portion thereof, and wherein a portion of said flange unit is positioned in said annular recess.

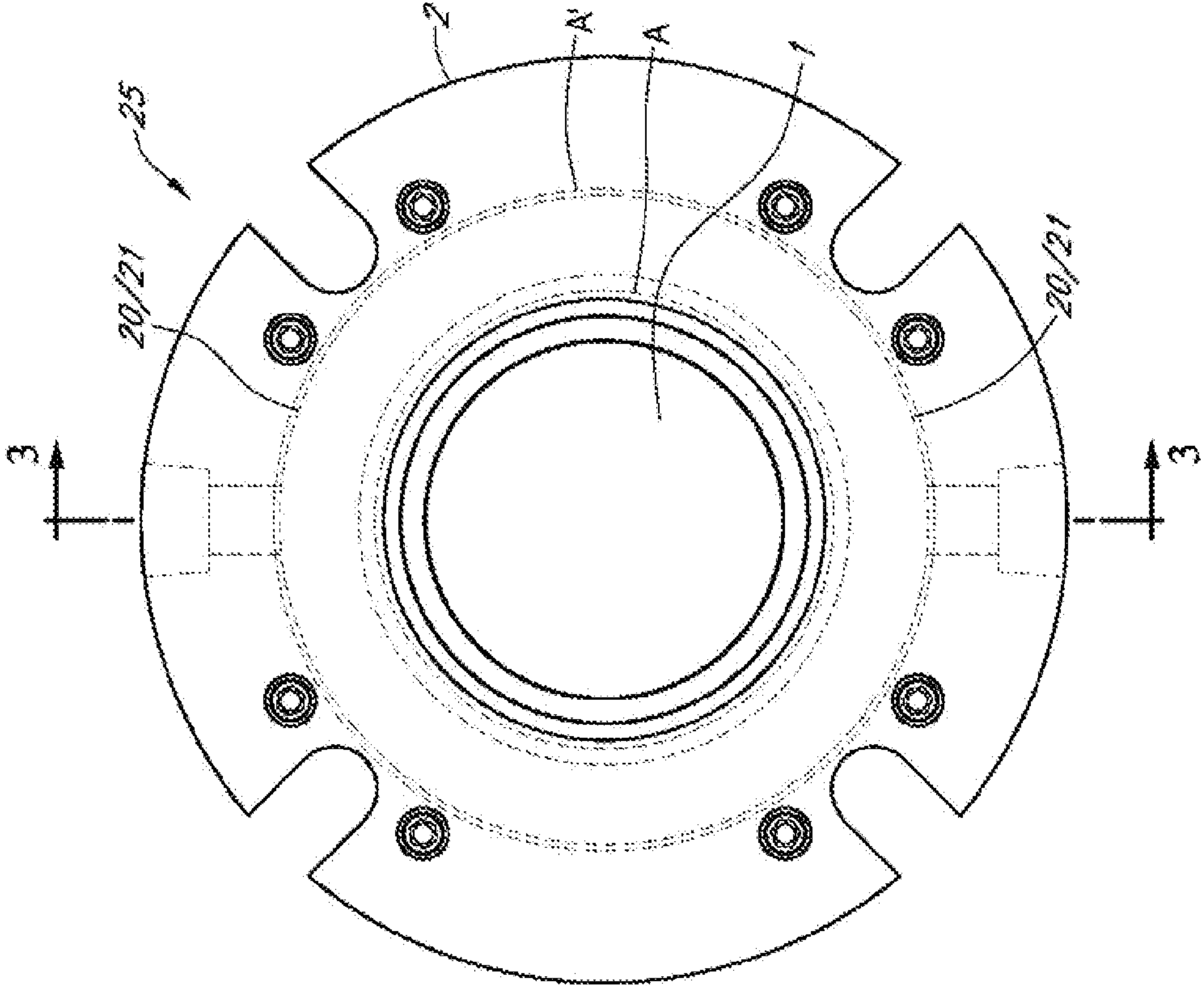


FIG. 2

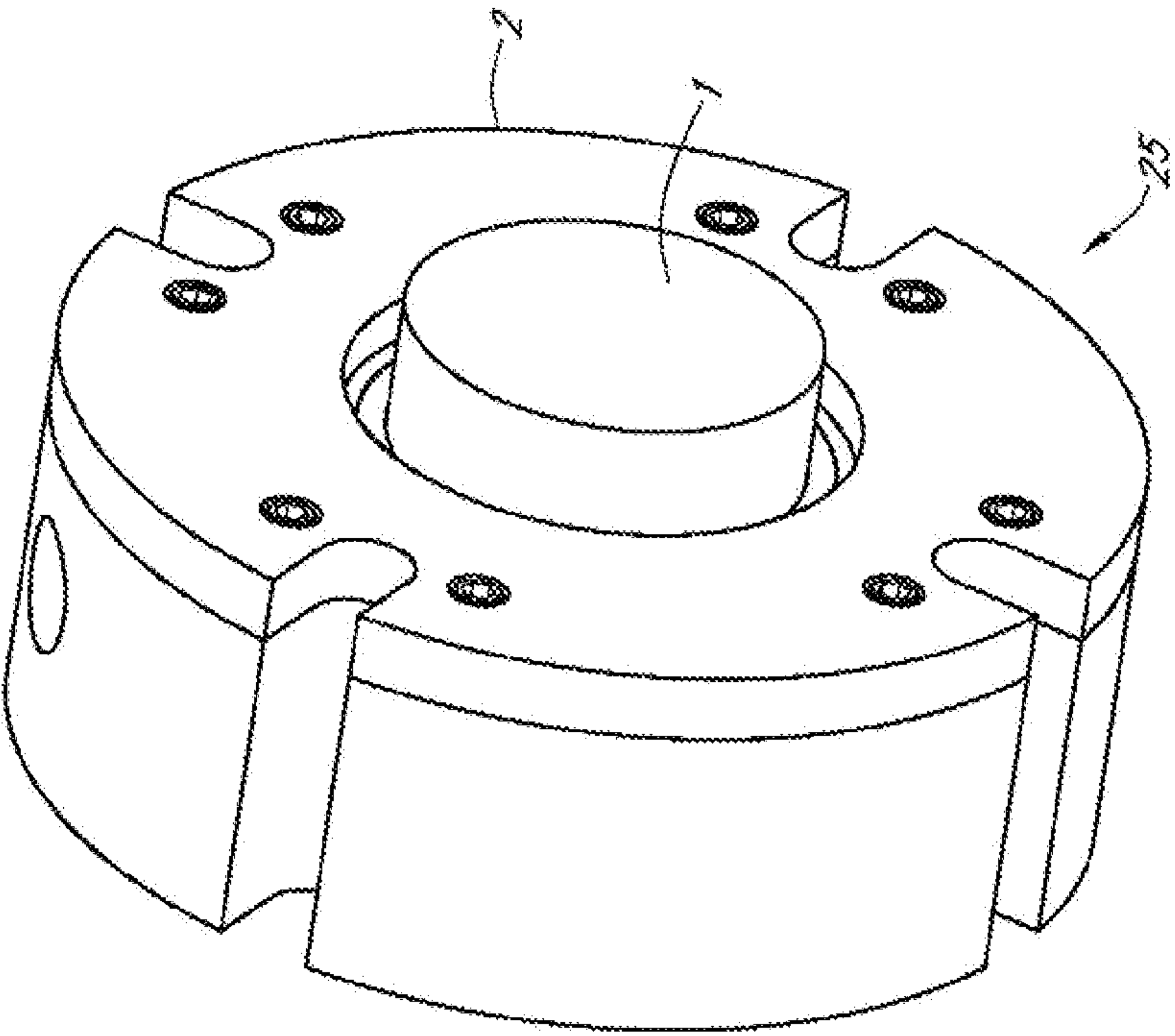
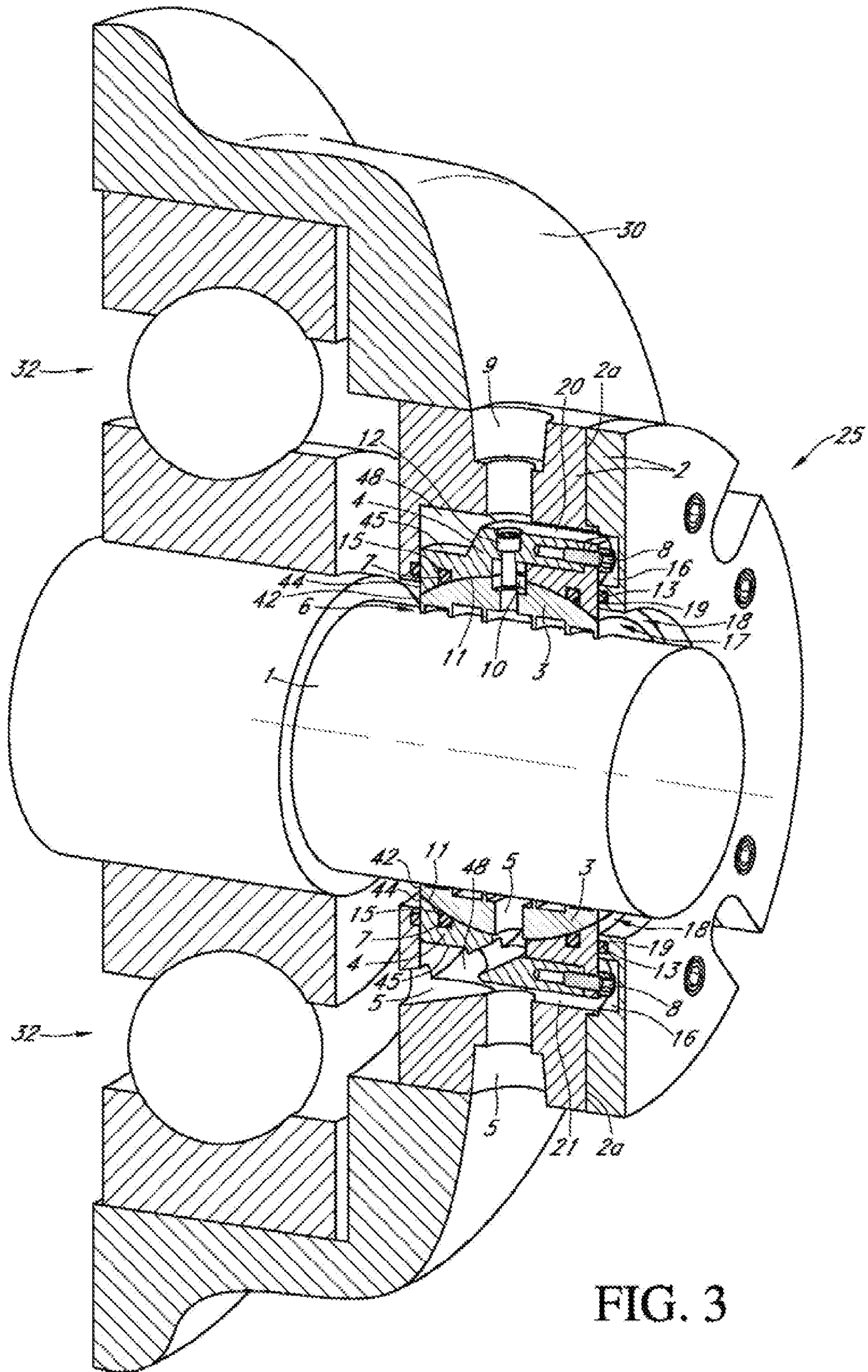


FIG. 1



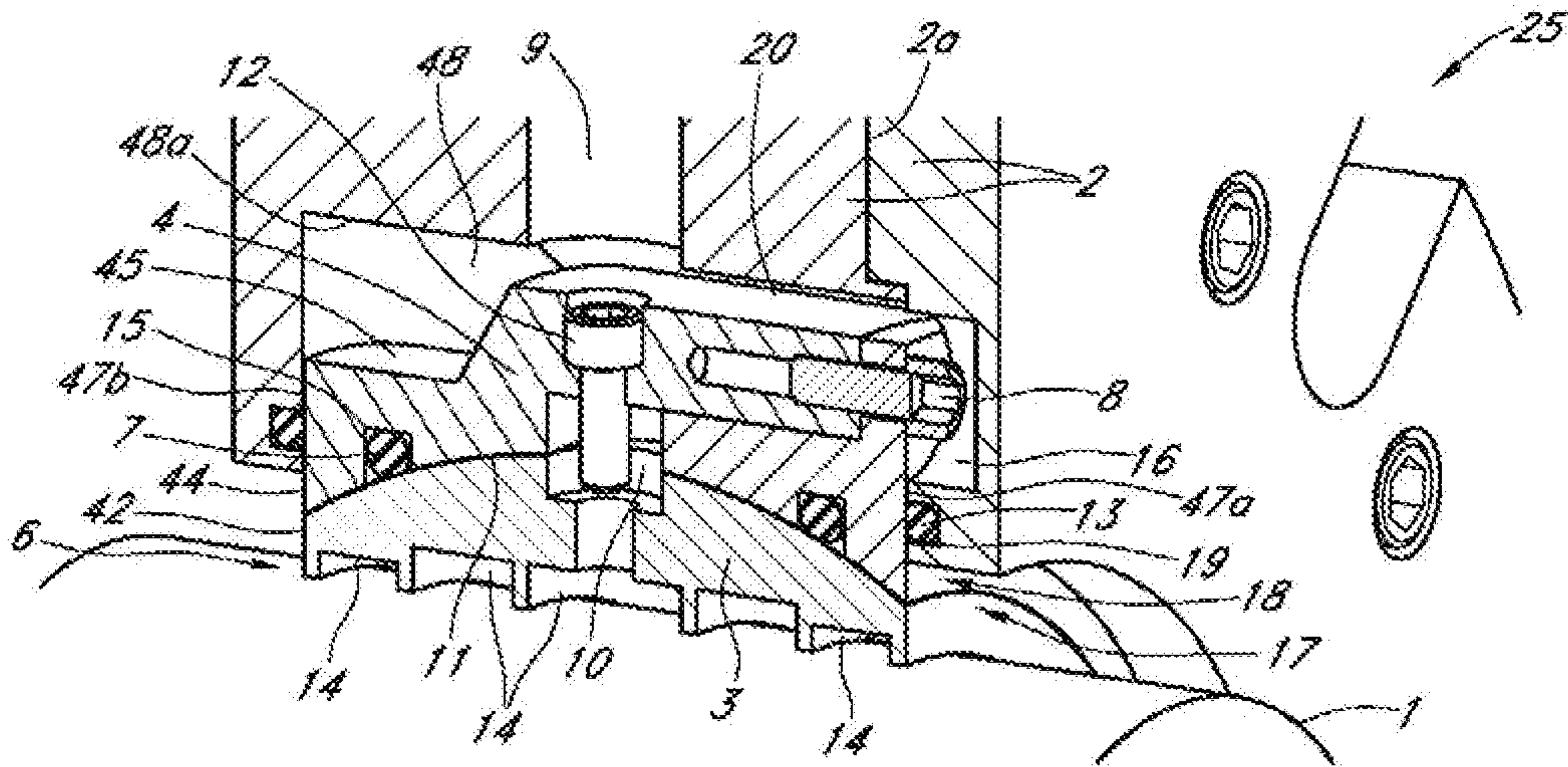


FIG. 3A

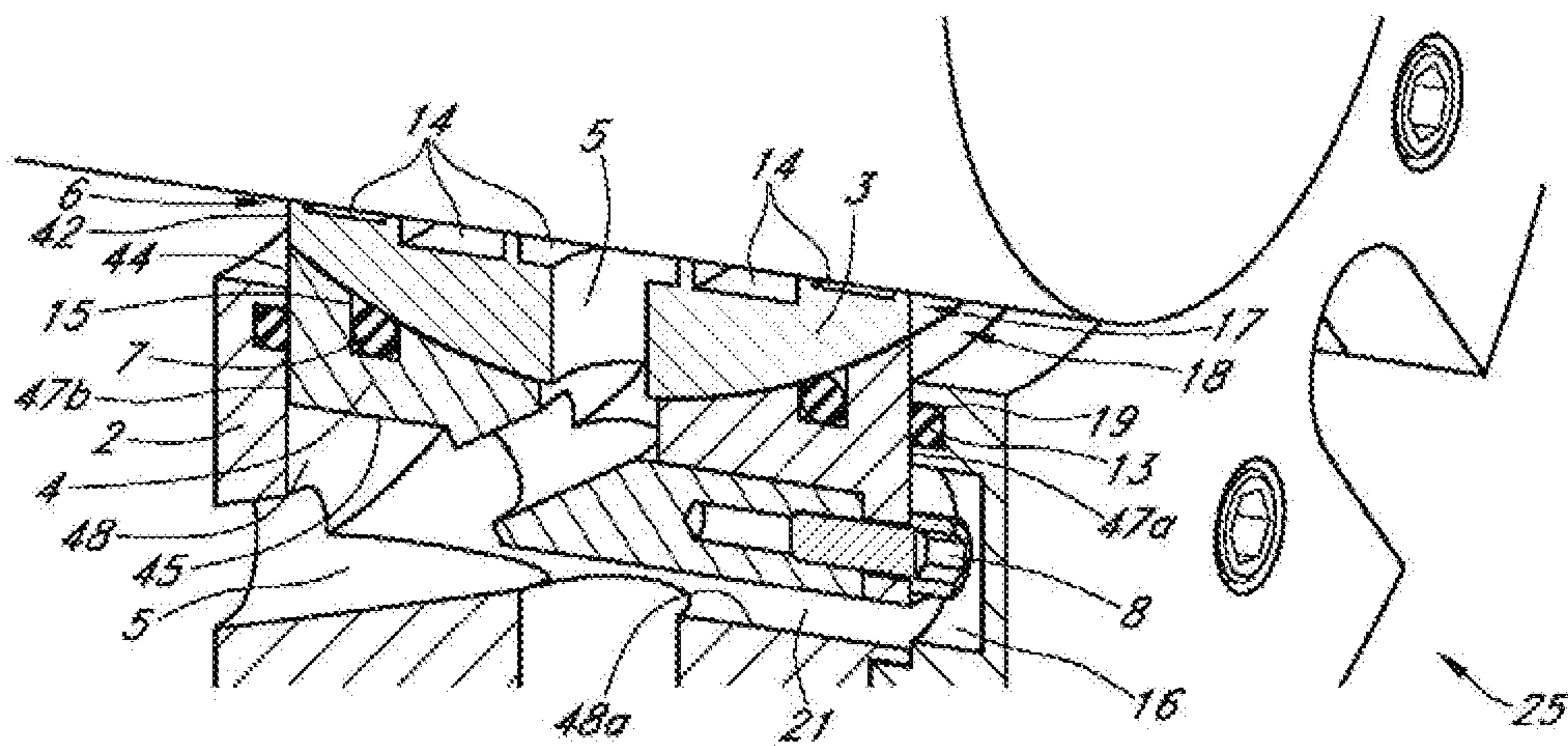


FIG. 3B

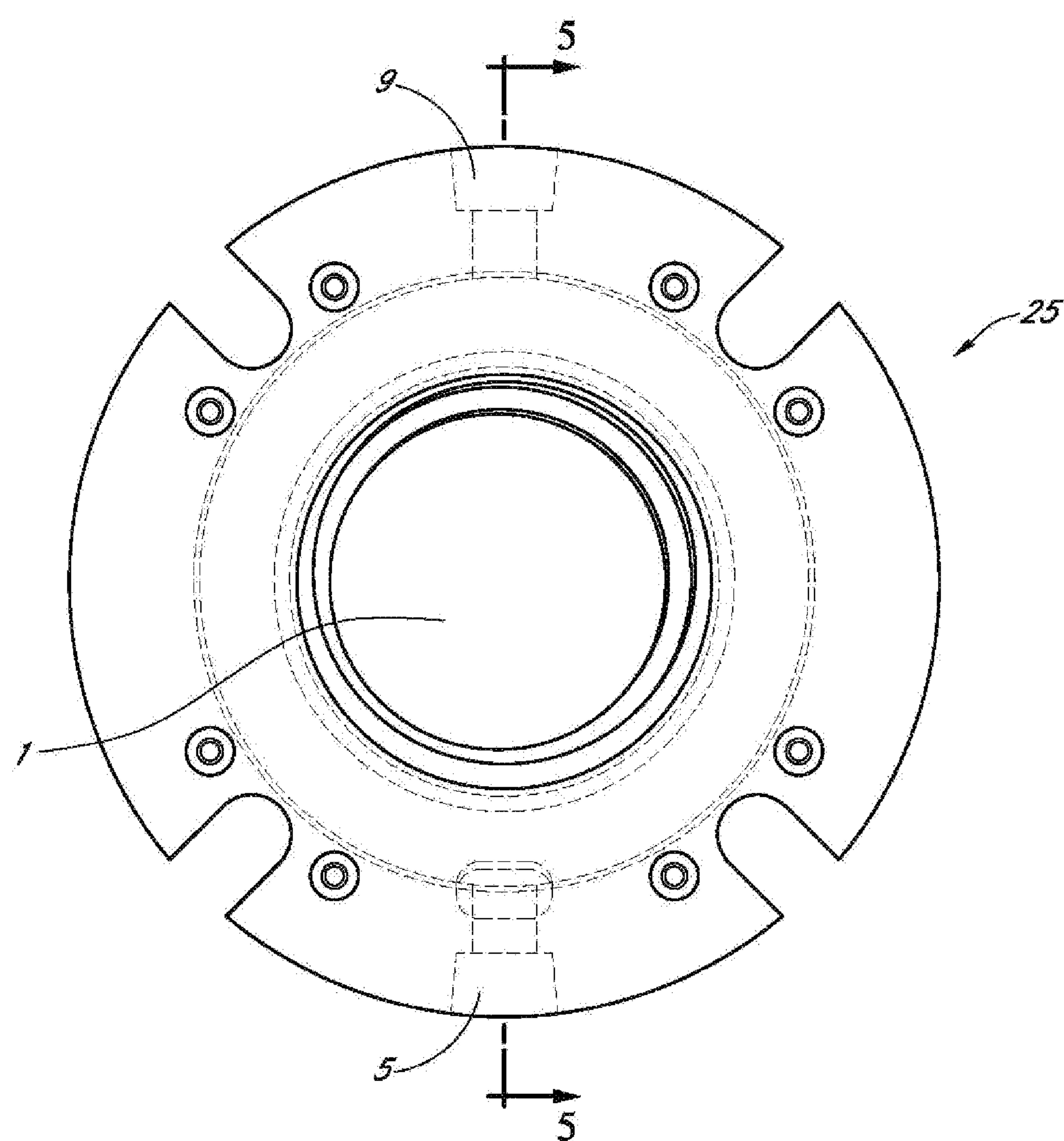


FIG. 4

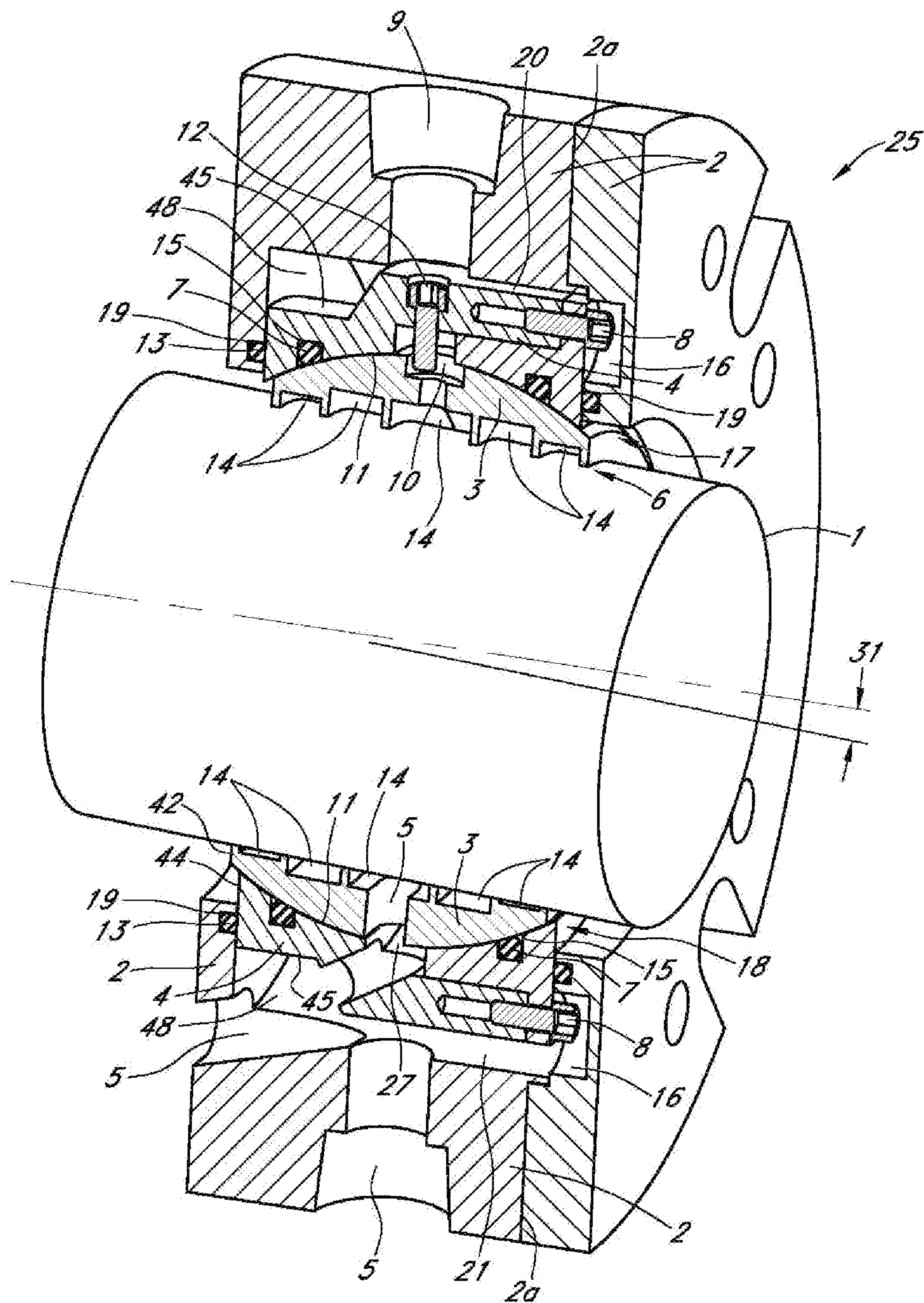


FIG. 5

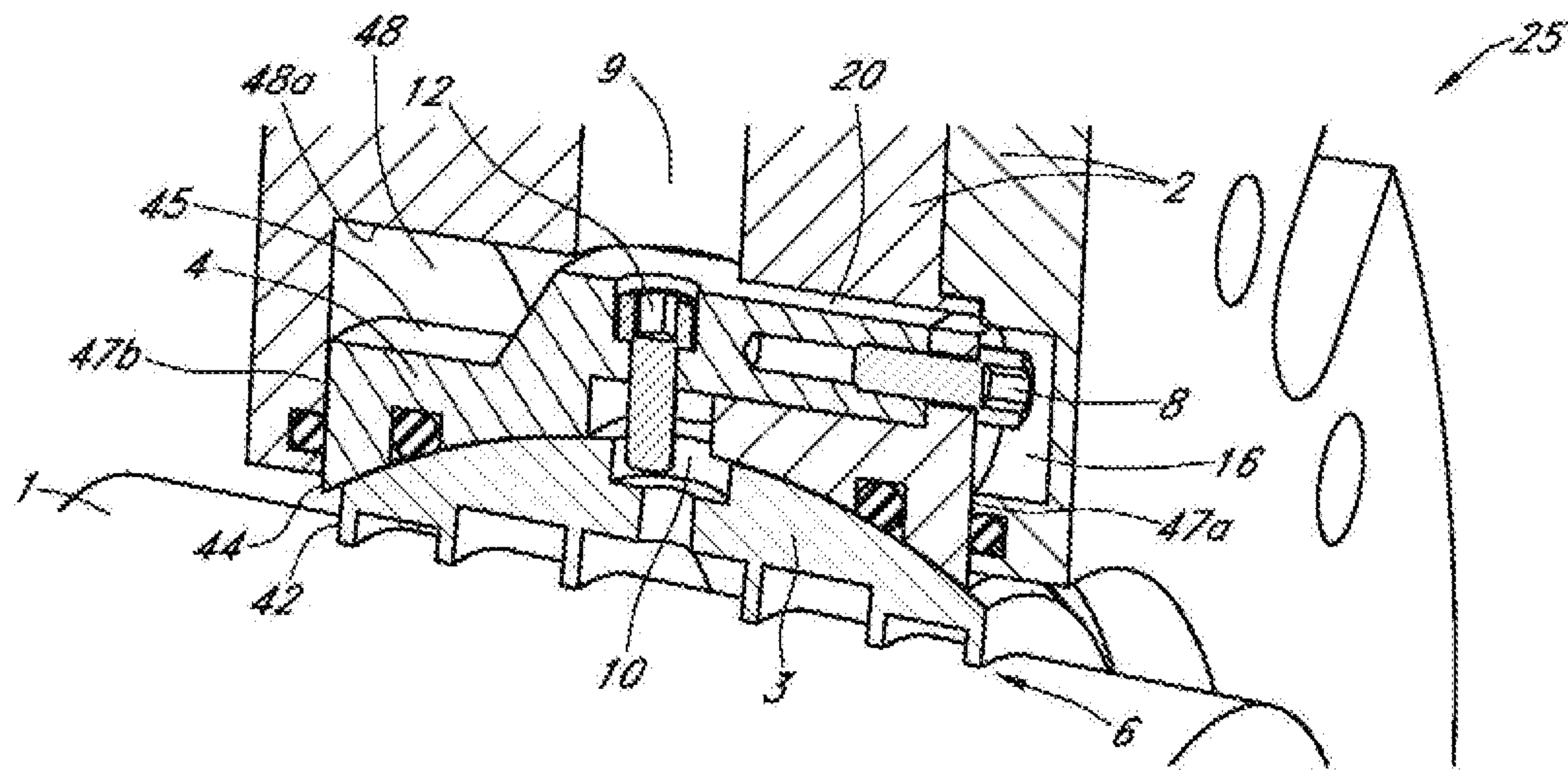


FIG. 5A

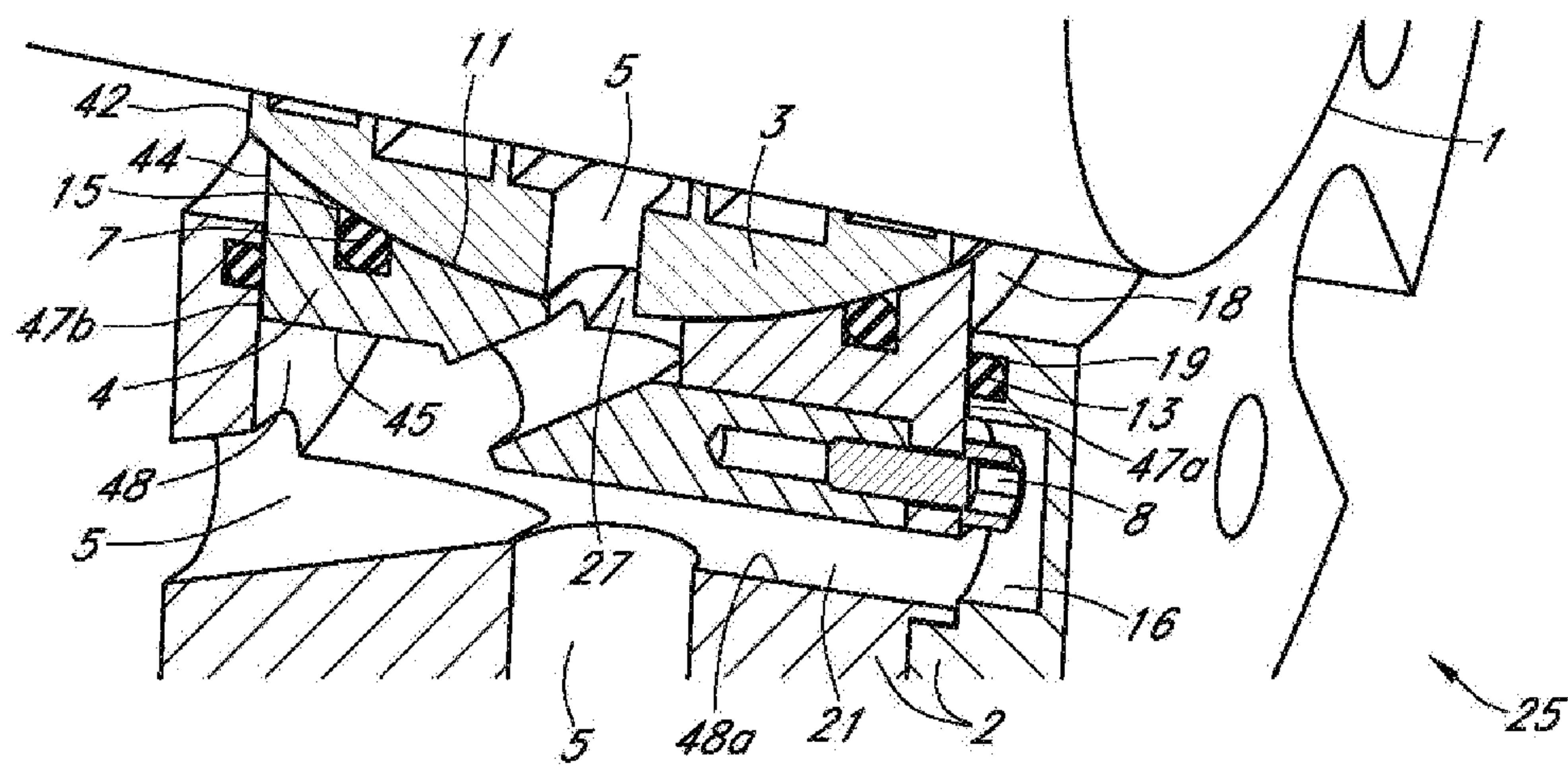


FIG. 5B

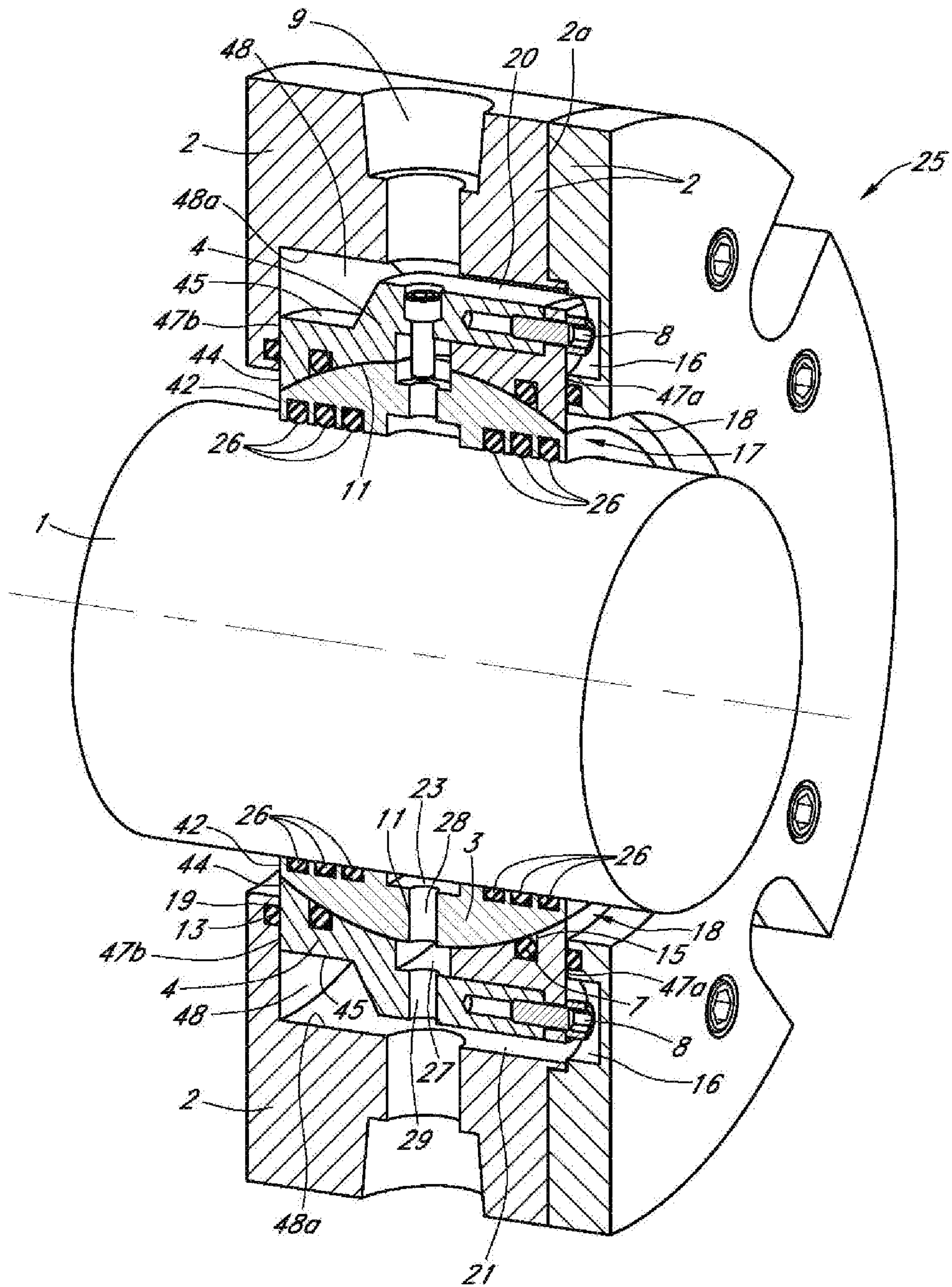


FIG. 6

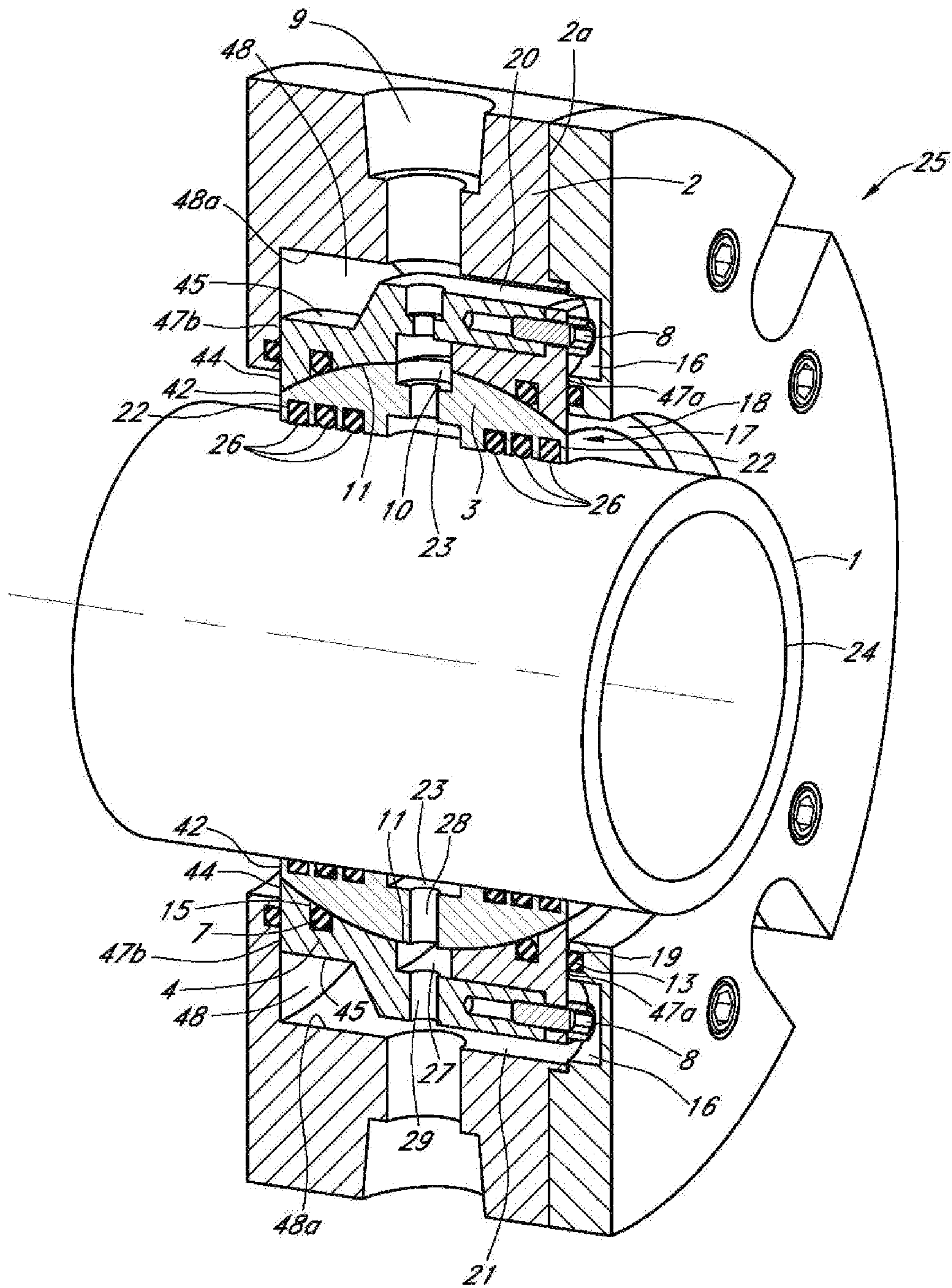


FIG. 7

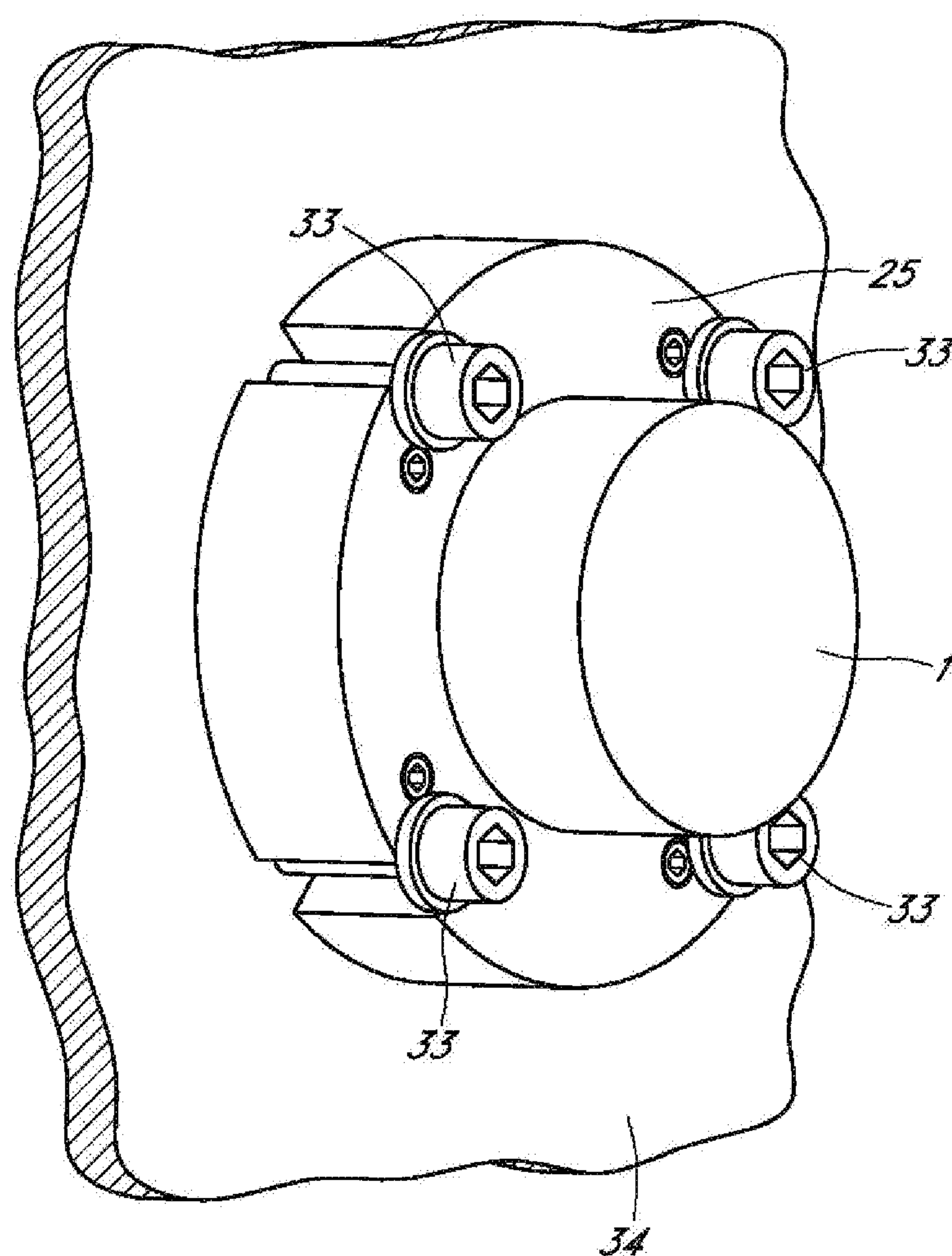


FIG. 8

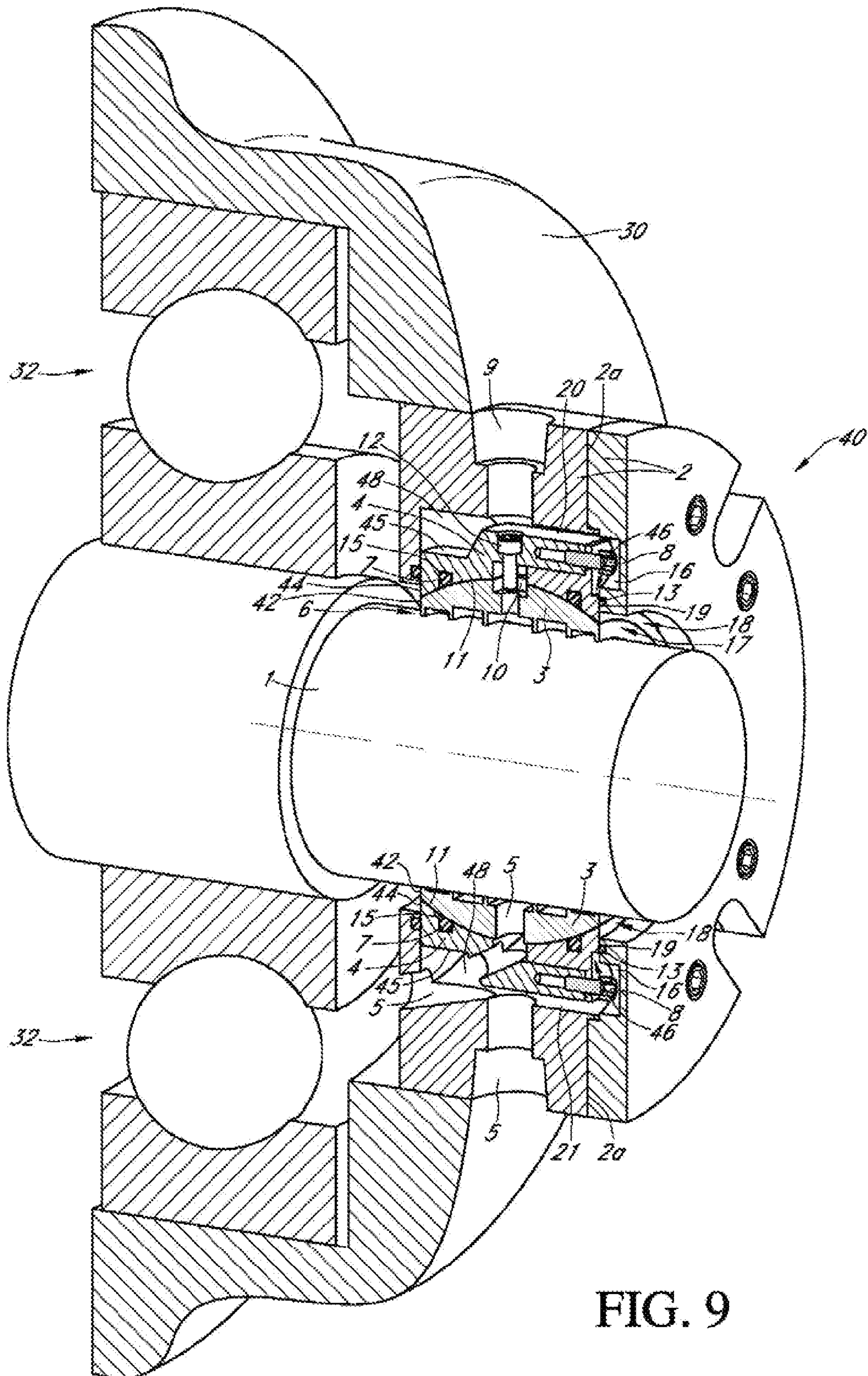


FIG. 9

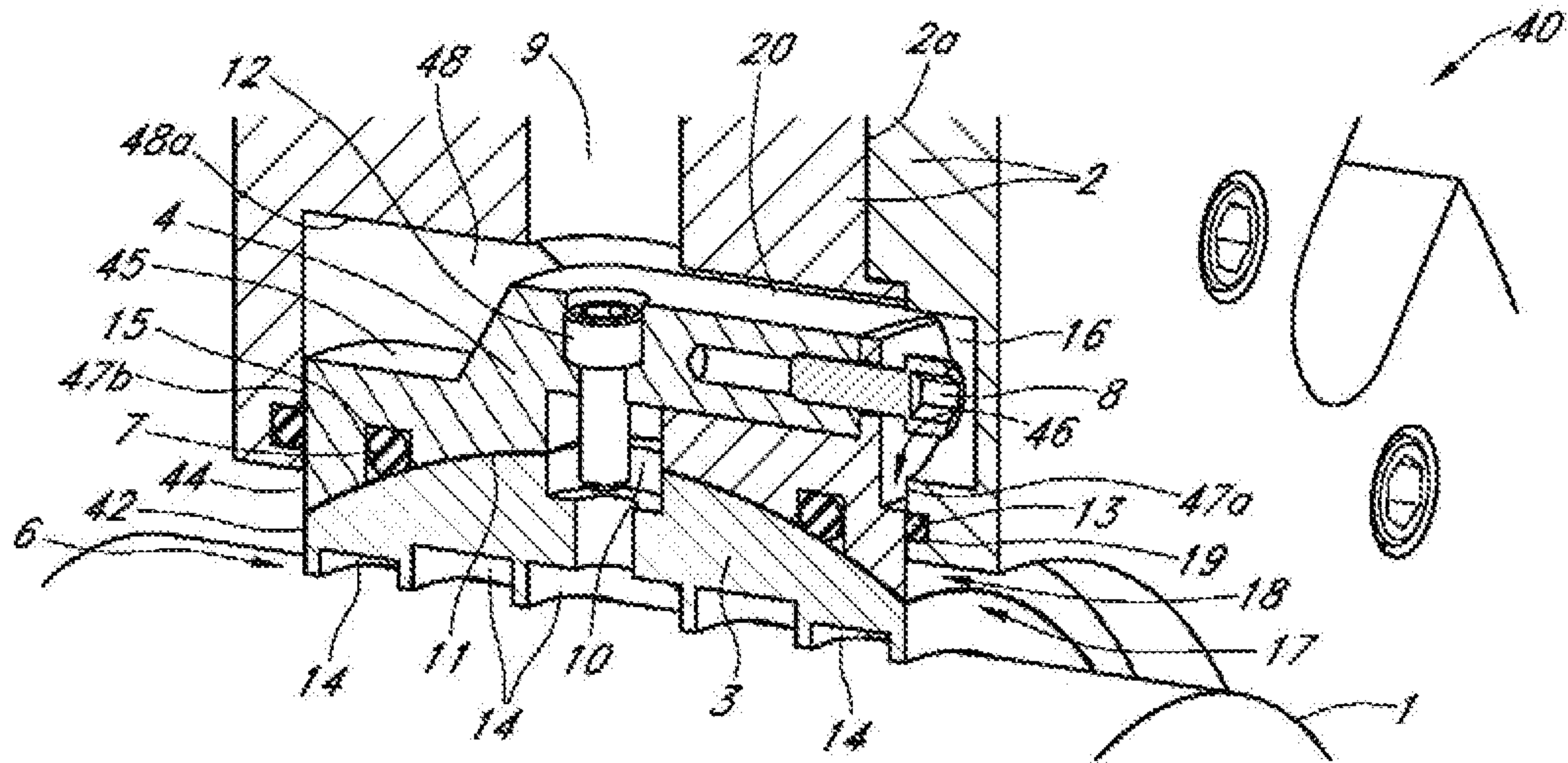


FIG. 9A

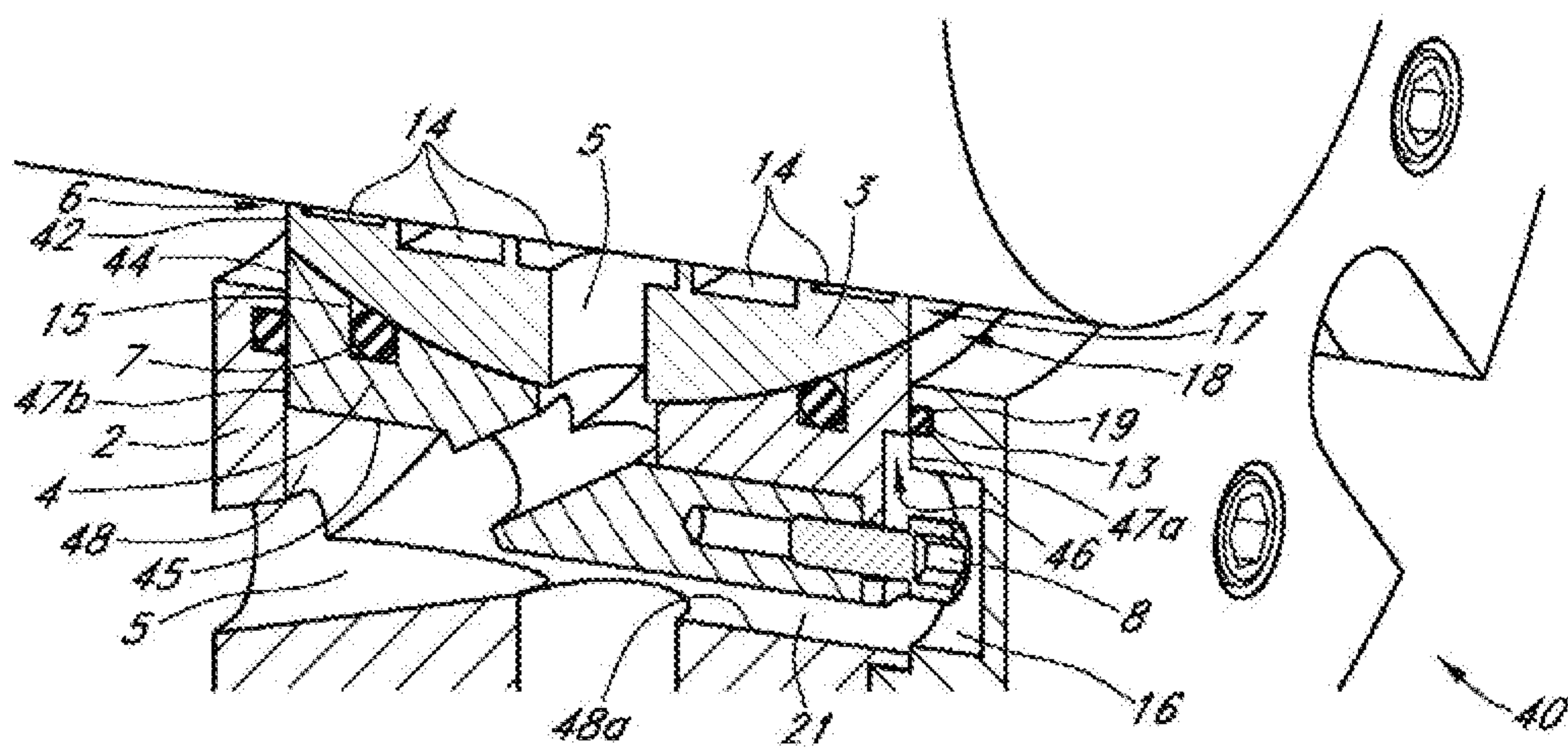


FIG. 9B

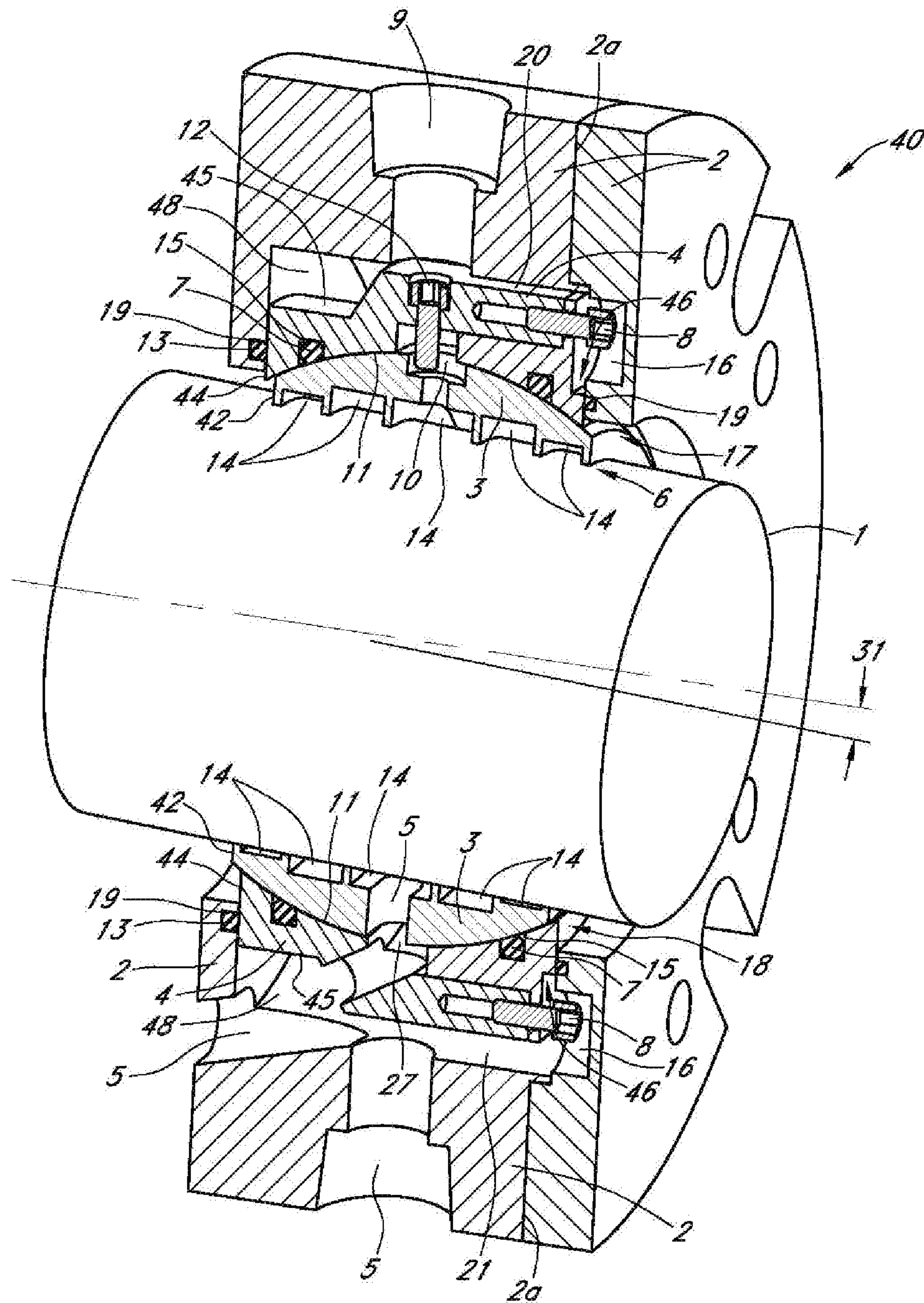


FIG. 10

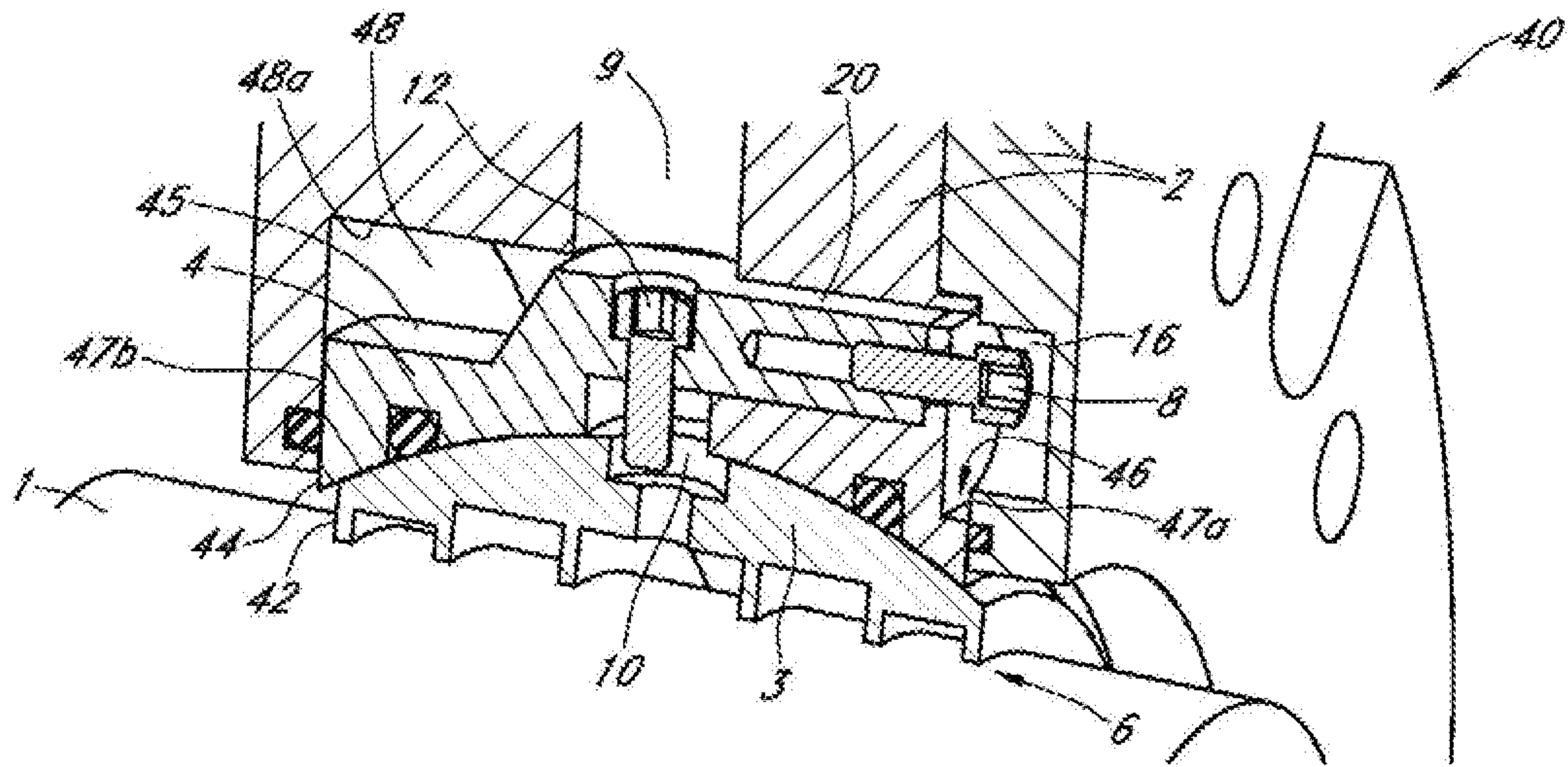


FIG. 10A

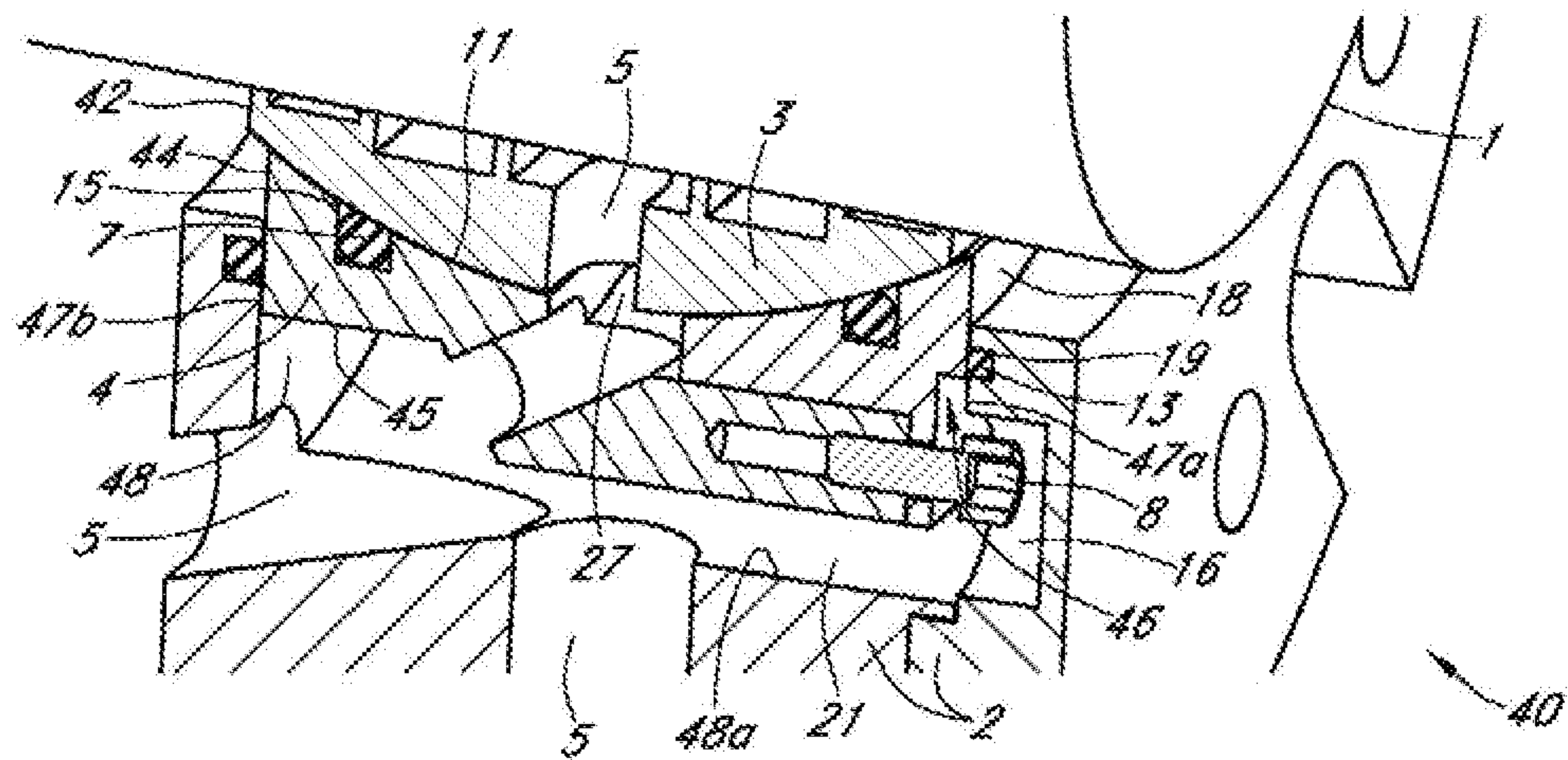


FIG. 10B

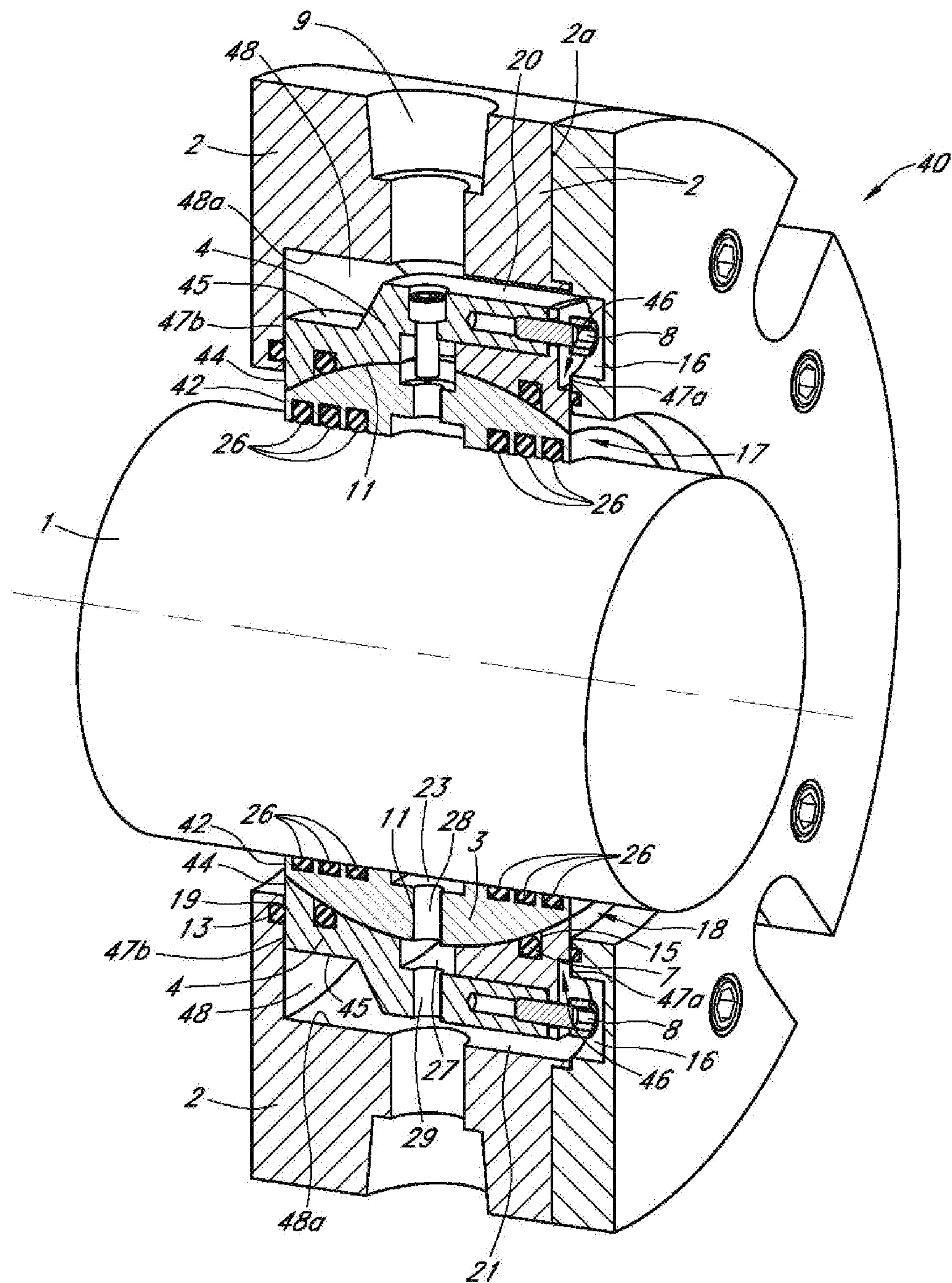


FIG. 11

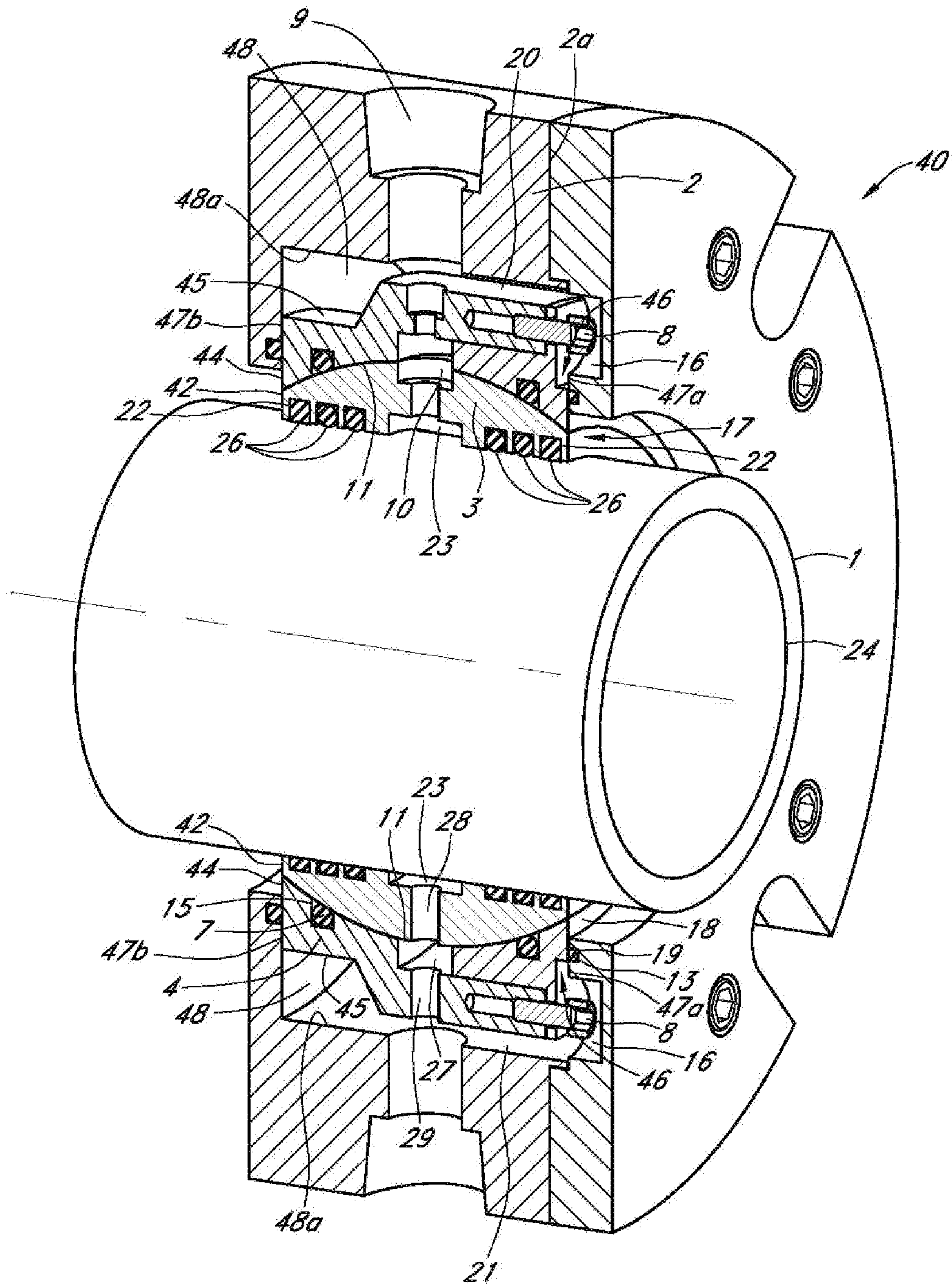


FIG. 12

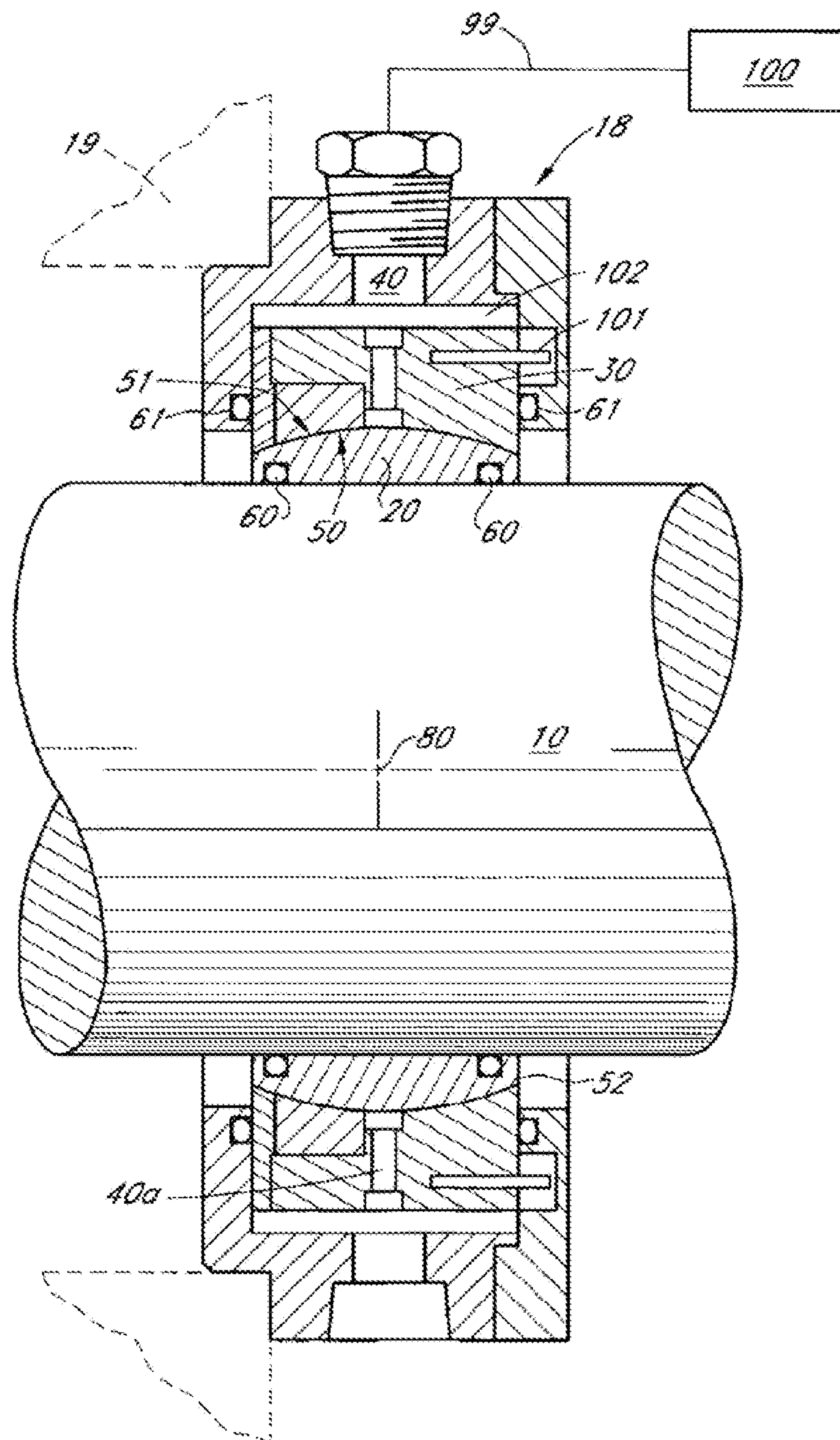
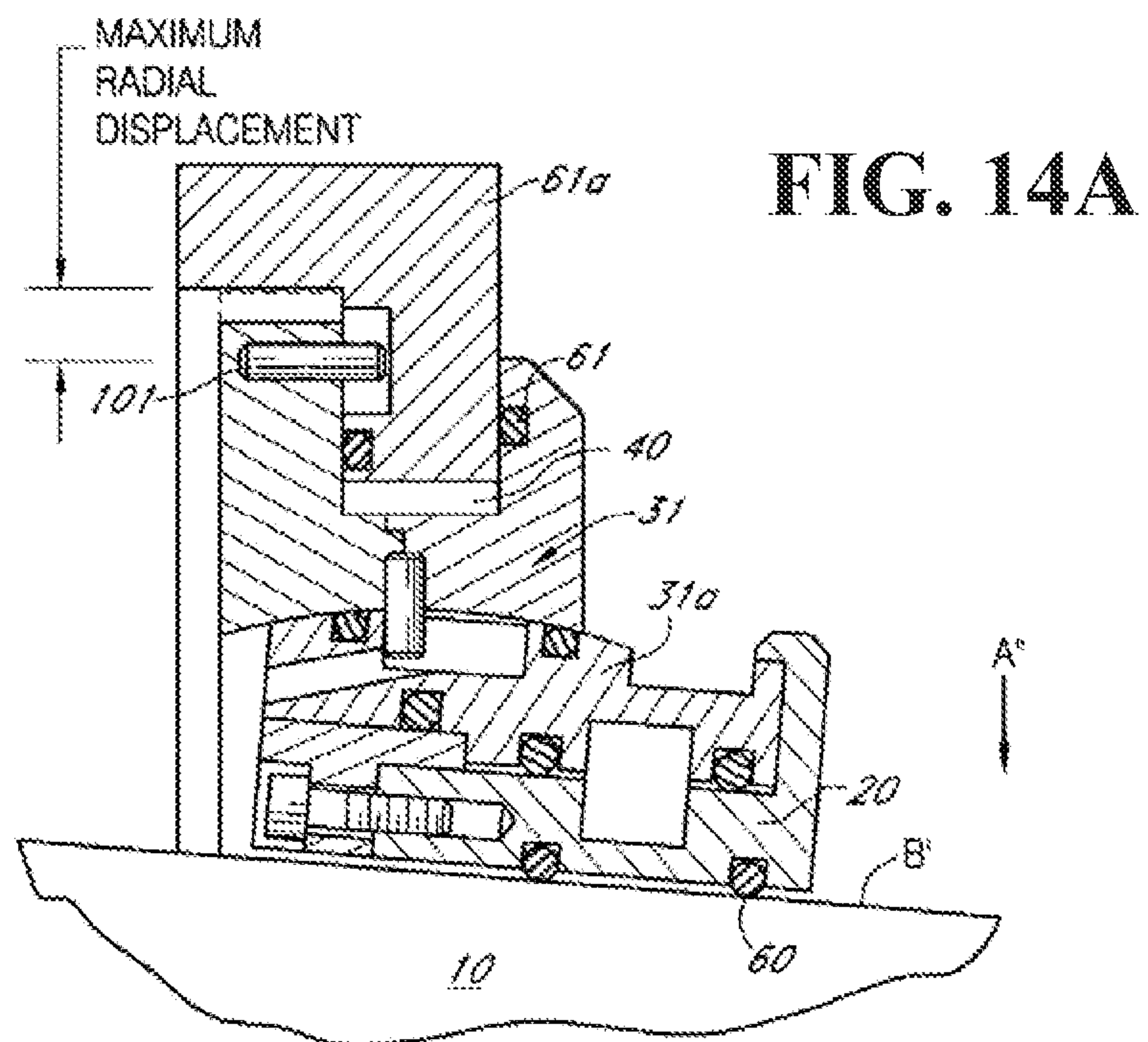
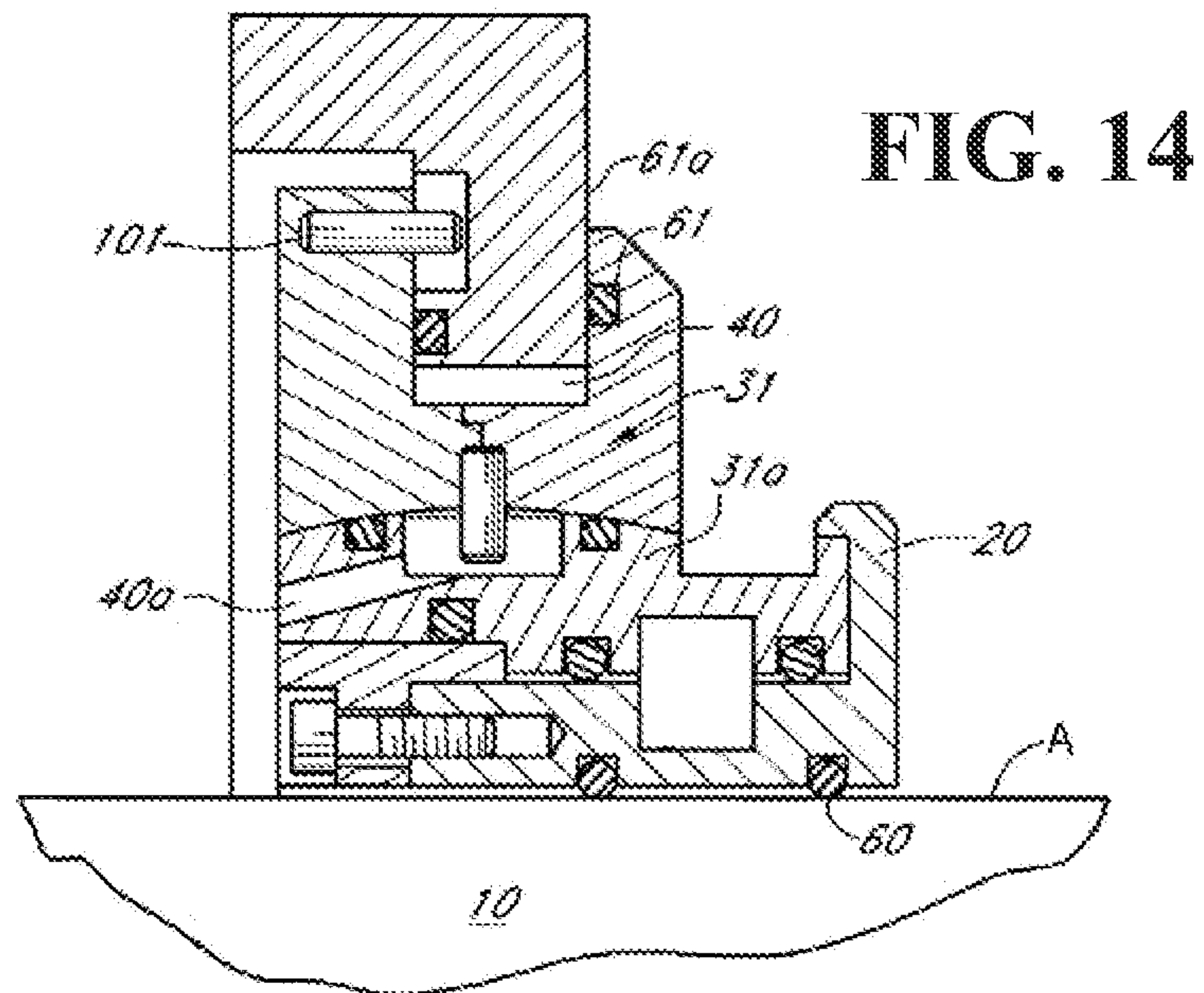


FIG. 13



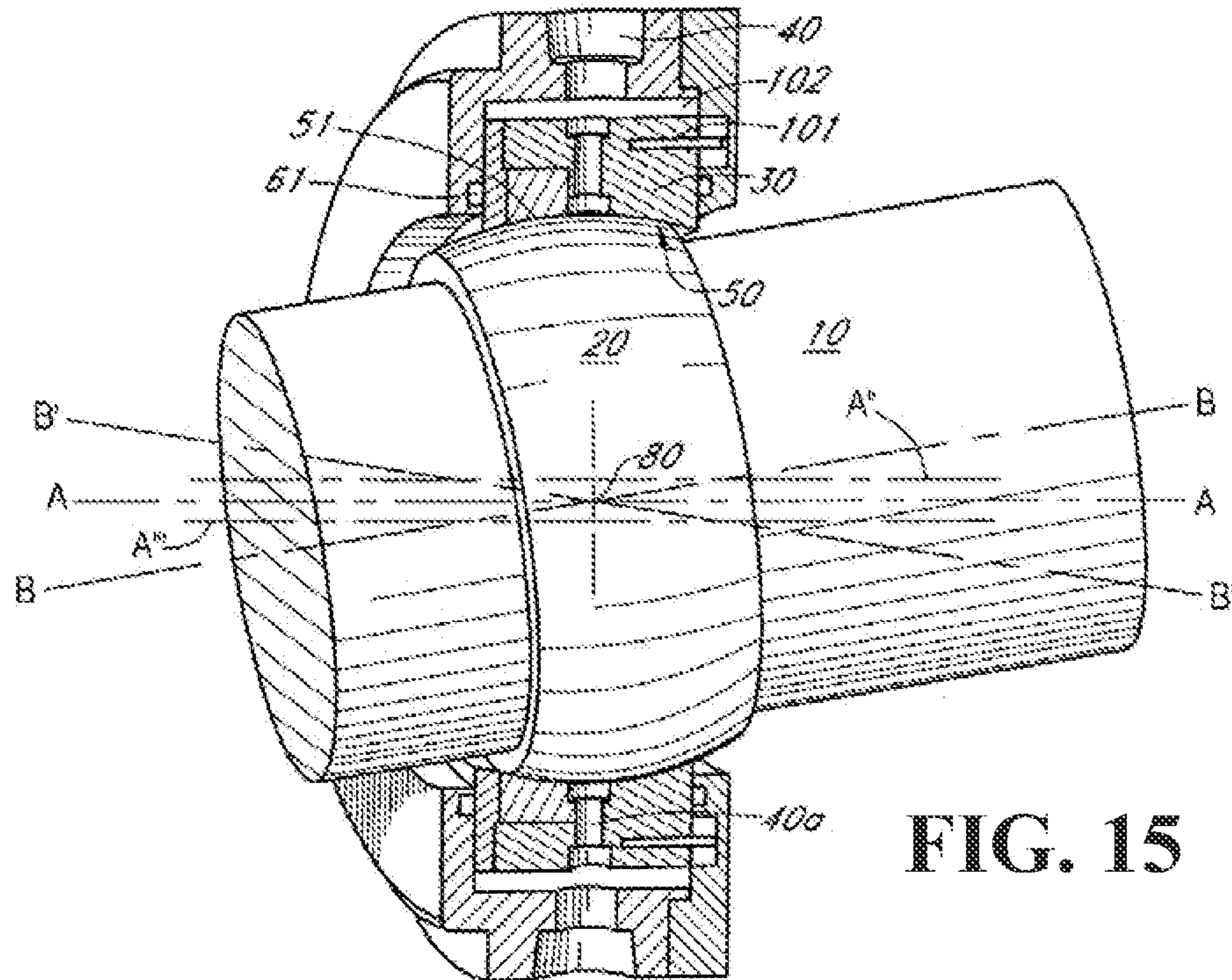
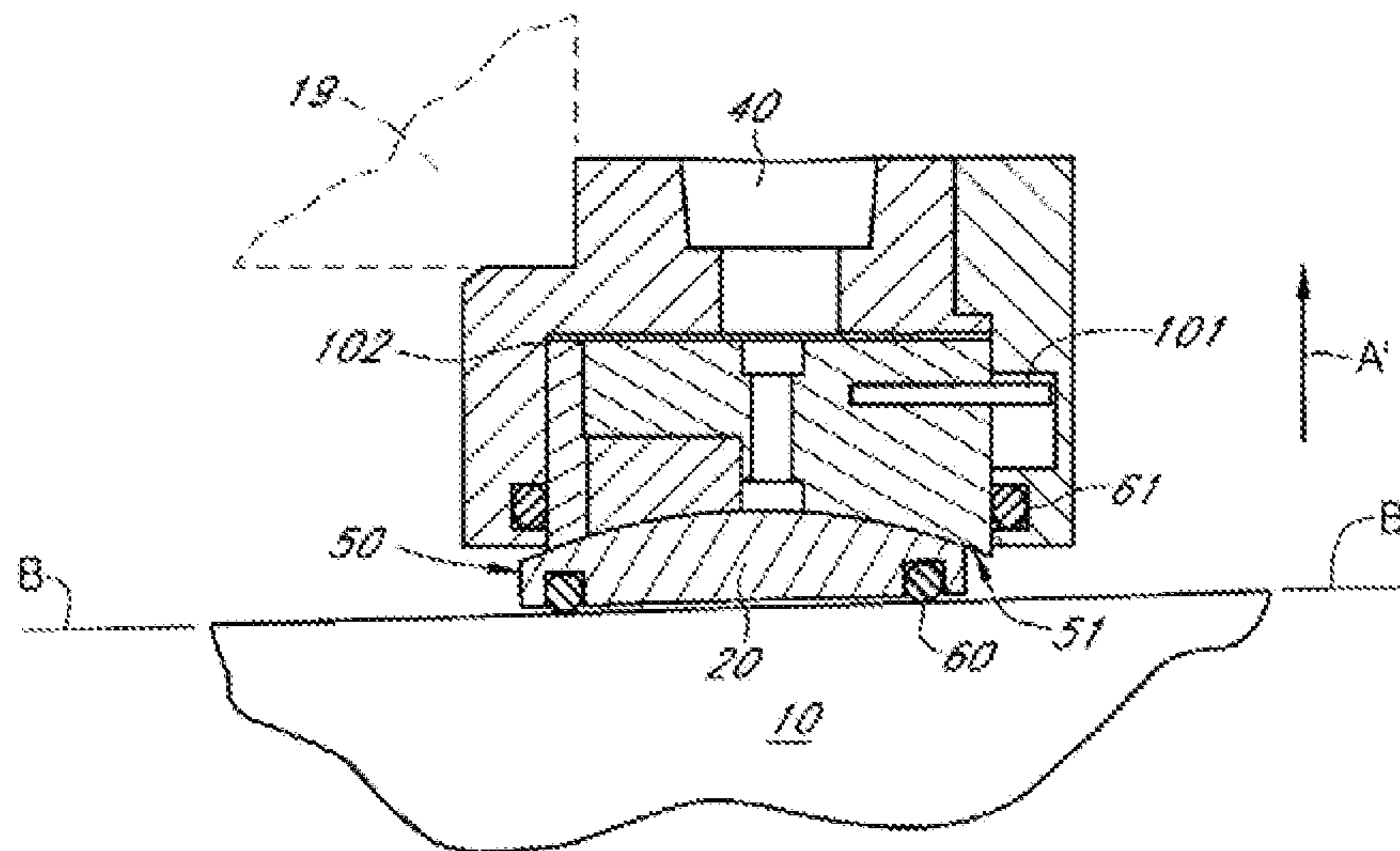


FIG. 15A



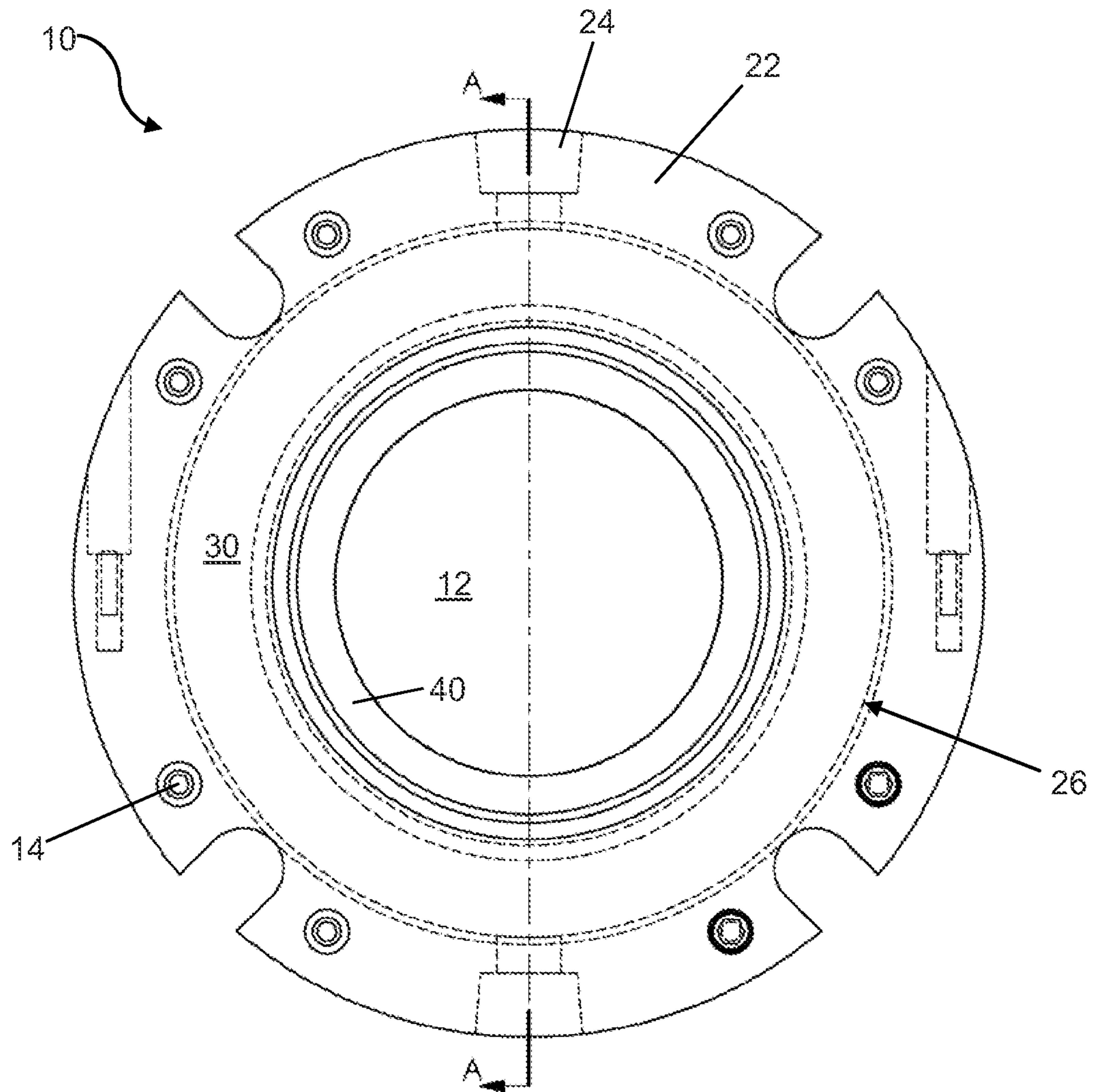


FIG. 16A

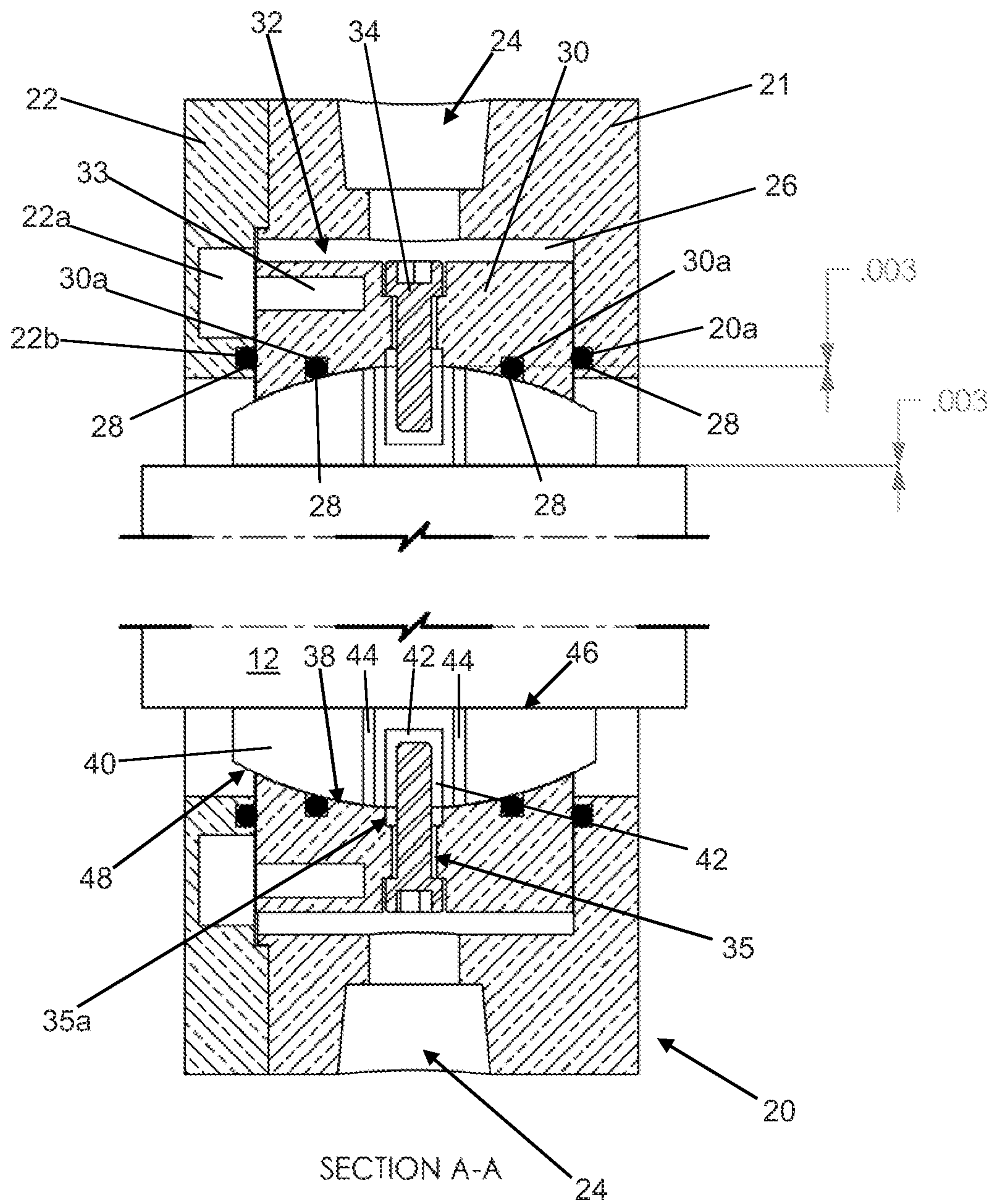


FIG. 16B

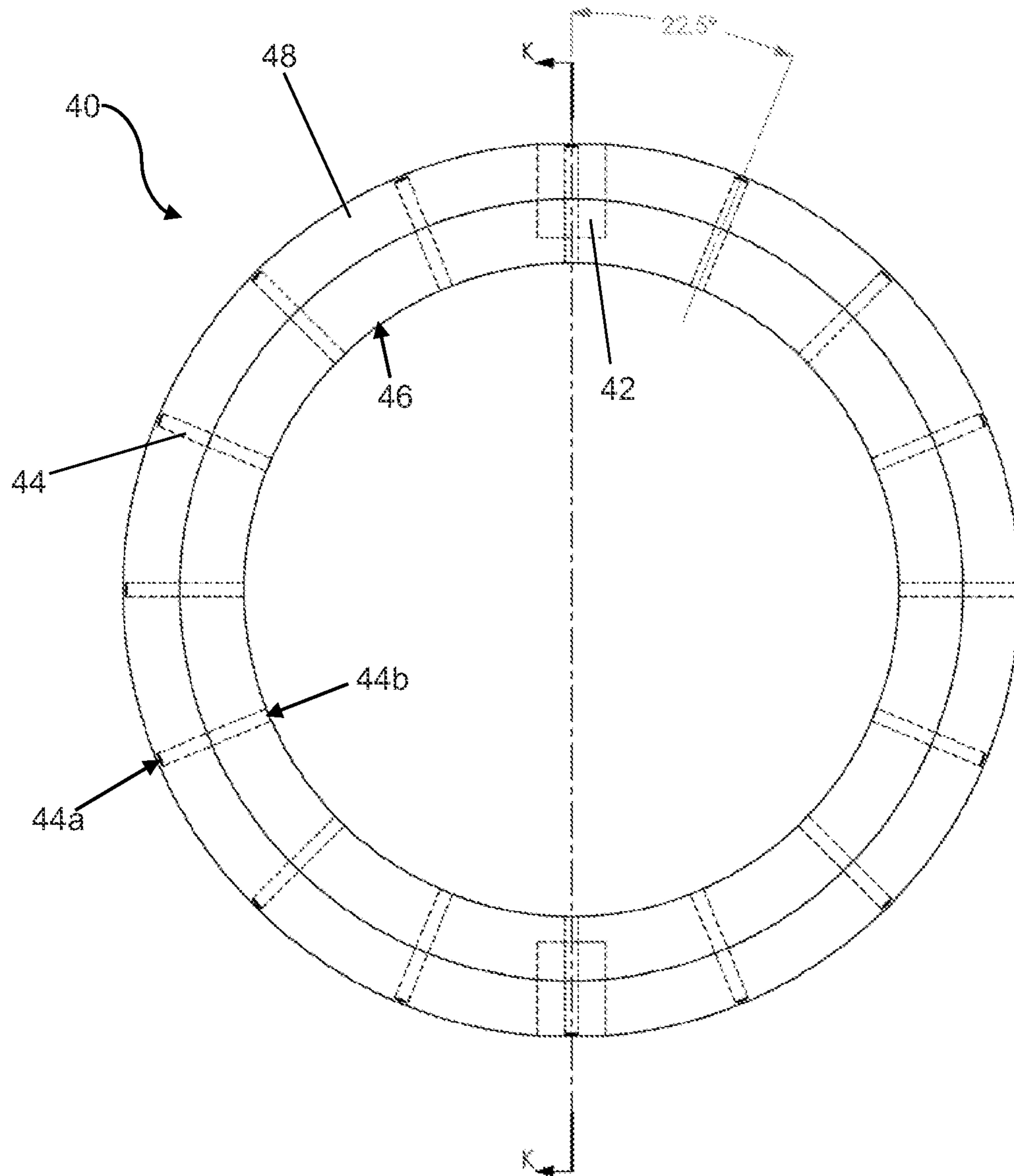
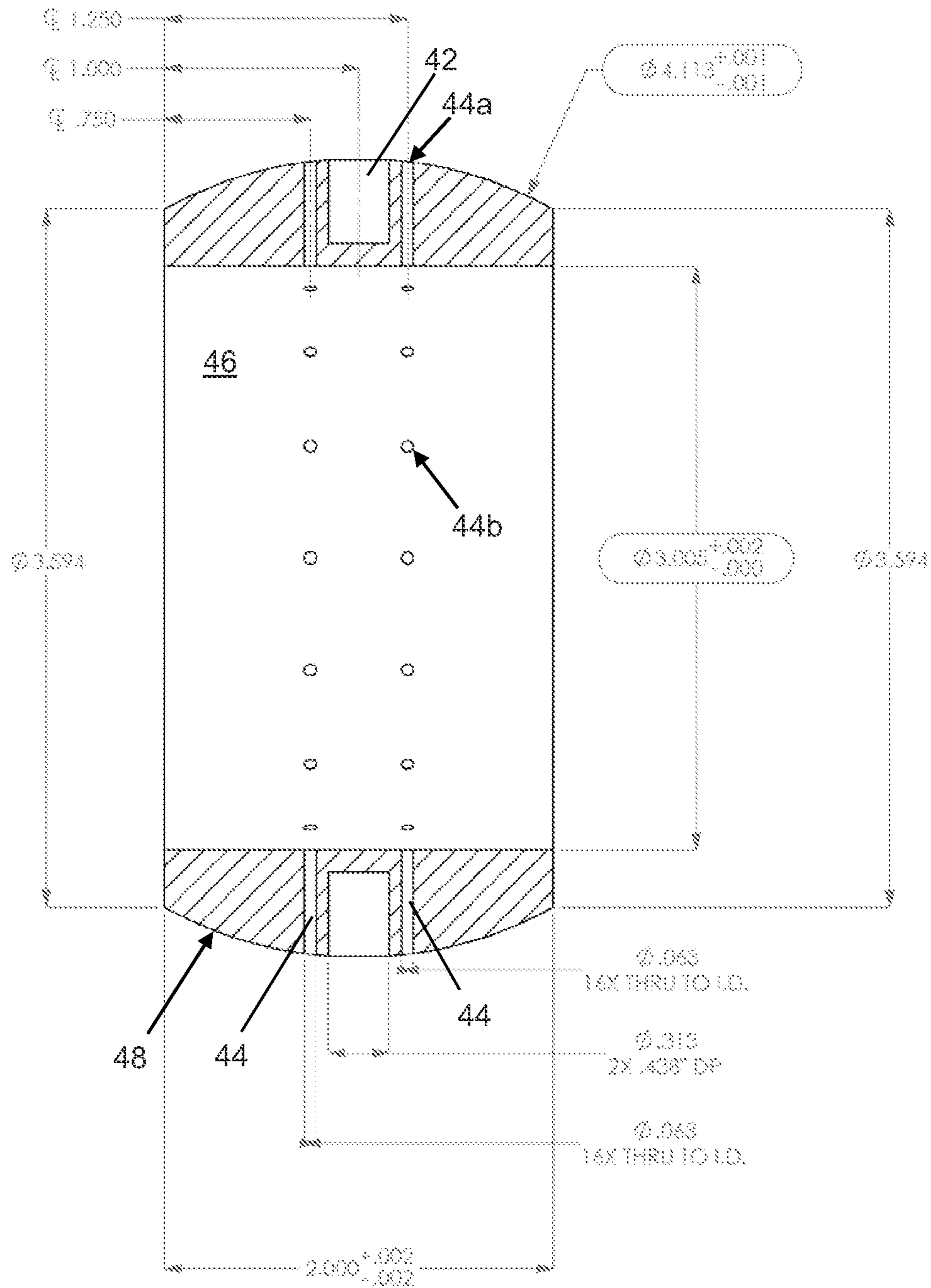


FIG. 17A



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FIG. 17B

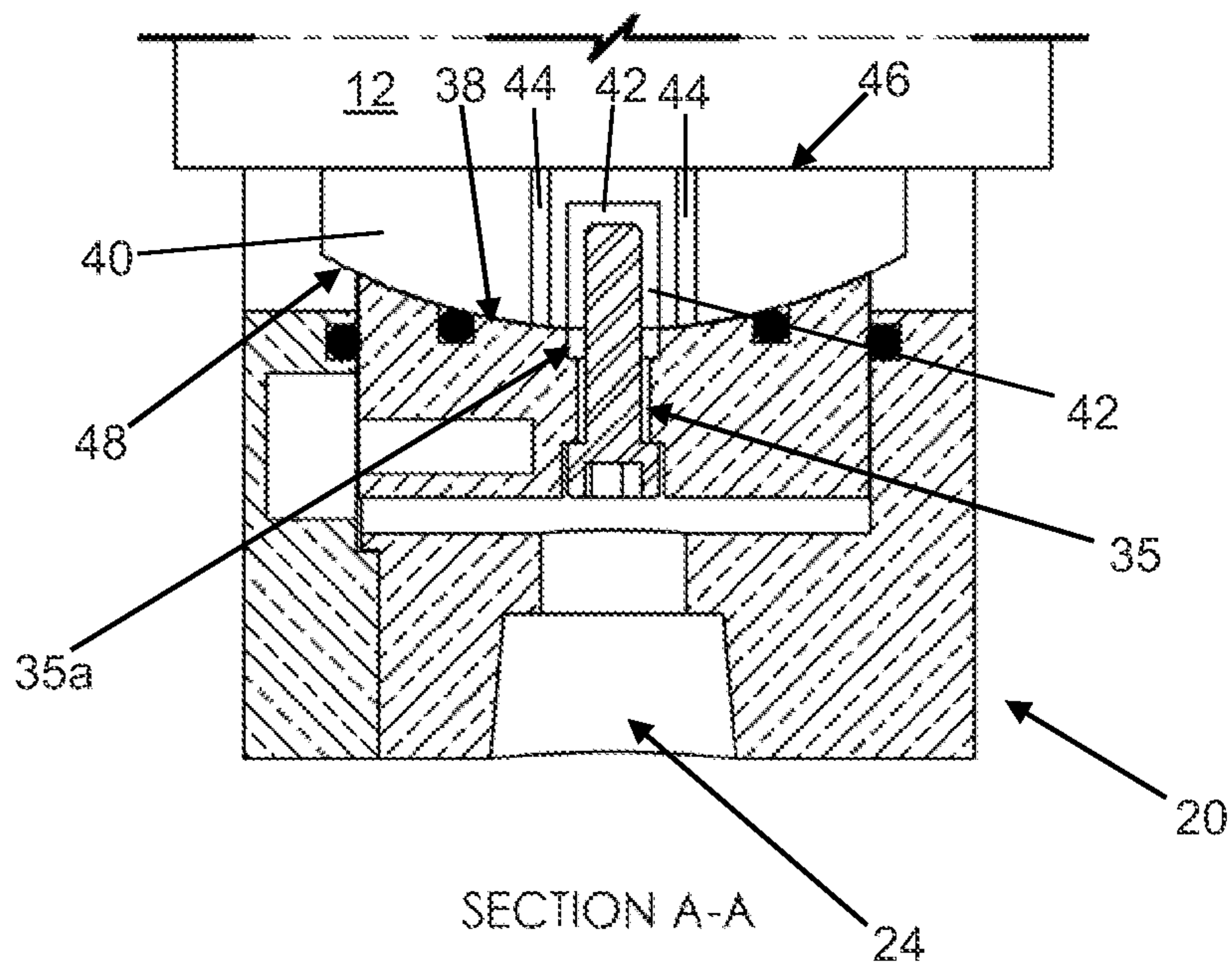
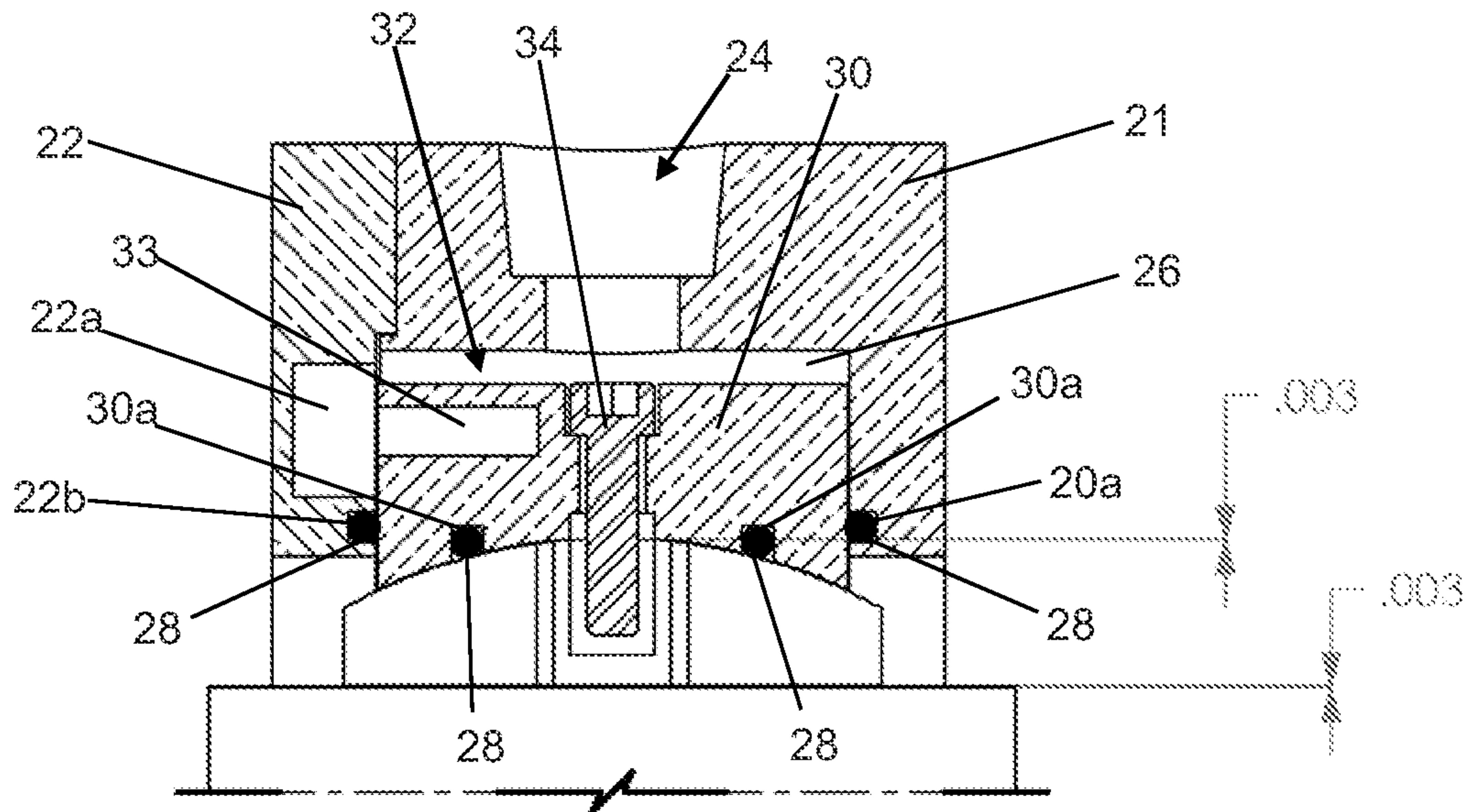


FIG. 16B