

[54] **AUTOMOTIVE SIDE DOOR LATCH**

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[52] U.S. Cl. ... **292/216; 292/DIG. 24; 292/DIG. 25**

[51] Int. Cl.² **E05C 3/28**

[58] Field of Search..... 292/216, 280, DIG. 24,
292/DIG. 25

[56] **References Cited**

UNITED STATES PATENTS

3,312,489	4/1967	Trudeau.....	292/48
3,666,305	5/1972	Schlichter.....	292/48
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Primary Examiner—Richard E. Moore

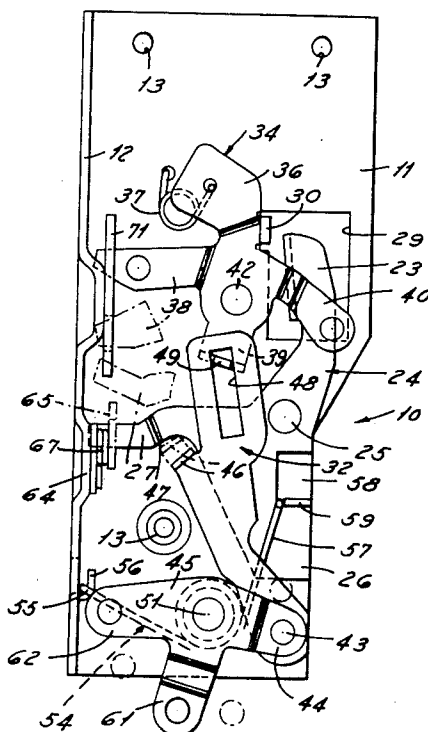
Attorney, Agent, or Firm—Whittemore, Hulbert &
Belknap

[57] **ABSTRACT**

A plate-like pawl blockingly engages between a pair of forked latch members, which are spring-biased toward one another to a position in which they encircle and latch a fixed automobile door keeper. The blocking part of the pawl is an integral plate component on a dogging arm of a heavy duty sheet metal stamping presenting three angularly divergent arms, the mentioned arm being undogged normally by a vertically

shiftable, intermittent latch operator, disconnecter and control slide when the lock is in an unlocked, non-freewheeling setting. To this end the slide has a laterally offset operating tab which is engageable for the purpose with an integral offset formation on a second de-dogging kick-off arm of the pawl, thus placing the latter in a latch releasing position. When the slide is cocked a bit transversely of its direction of longitudinal undogging shift, in a pivotal movement about a connection to an outside-operated bell crank, the slide overruns or free-wheels relative to the pawl arm's offset, hence disables the pawl from unlatching the lock. For the purpose of thus cocking the slide, the latter is provided with an elongated slot which receives a laterally offset tang on one arm of a pivoted multi-arm locking lever whereby, upon a swing of that lever in a pawl-disabling and locking direction, the slide is shifted or cocked to its position to overrun the pawl arm offset. Otherwise, the tab of the slide operates to undog the pawl without interference from the locking lever tang, since the slide's slot receiving said tang is elongated, in the unlocked setting, on the direct line of shift of the operator and disconnect slide, hence is idly overrun by the latter. The locking lever is selectively operated by inside remote control and key-operated lever connections. The outside operator is a bell crank type, on an arm of which a bottom portion of the disconnect slide has a fixed, rivet-type pivotal operating connection. An automatic feature is also incorporated to enable the door to be locked without key from the outside.

14 Claims, 7 Drawing Figures



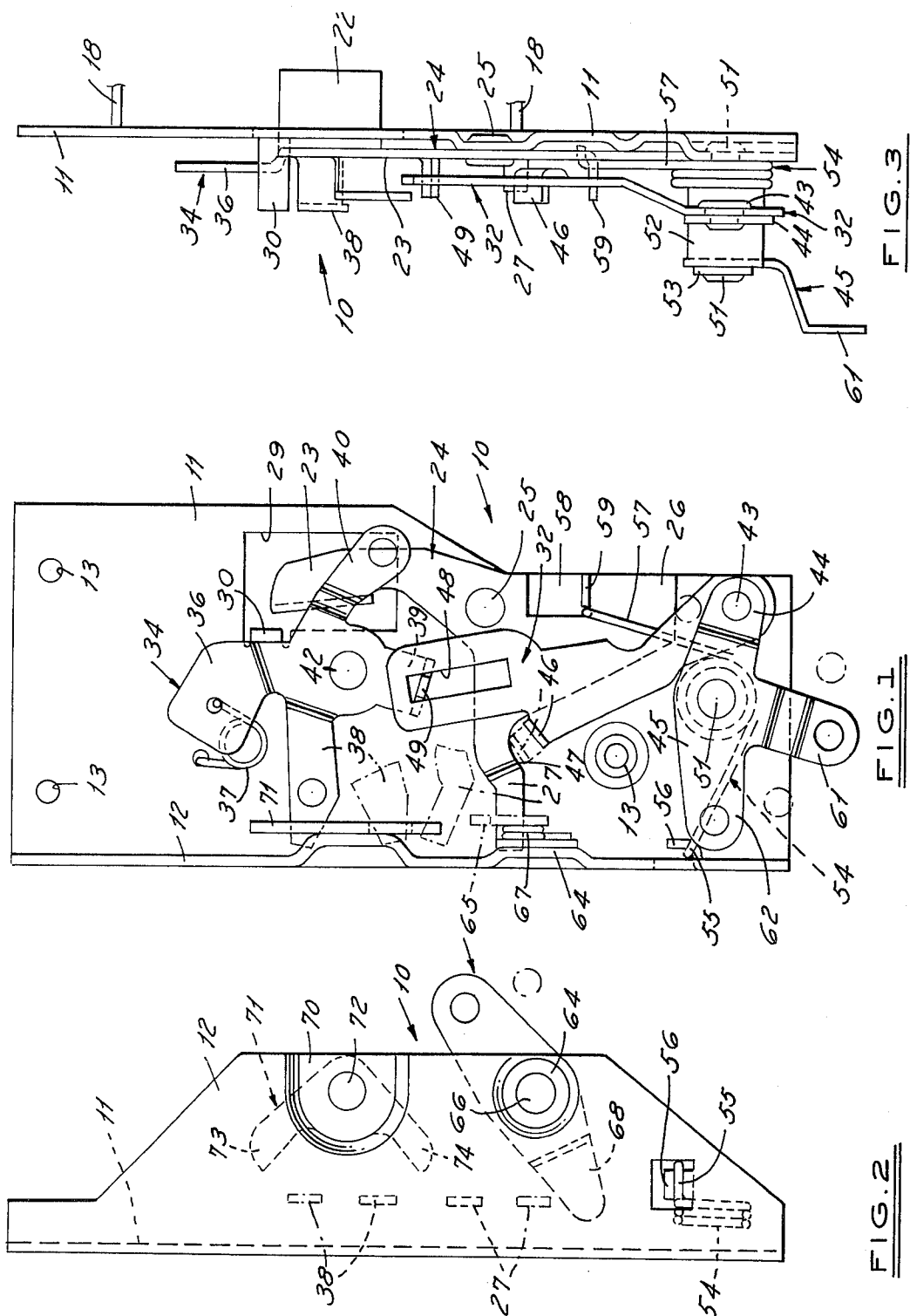
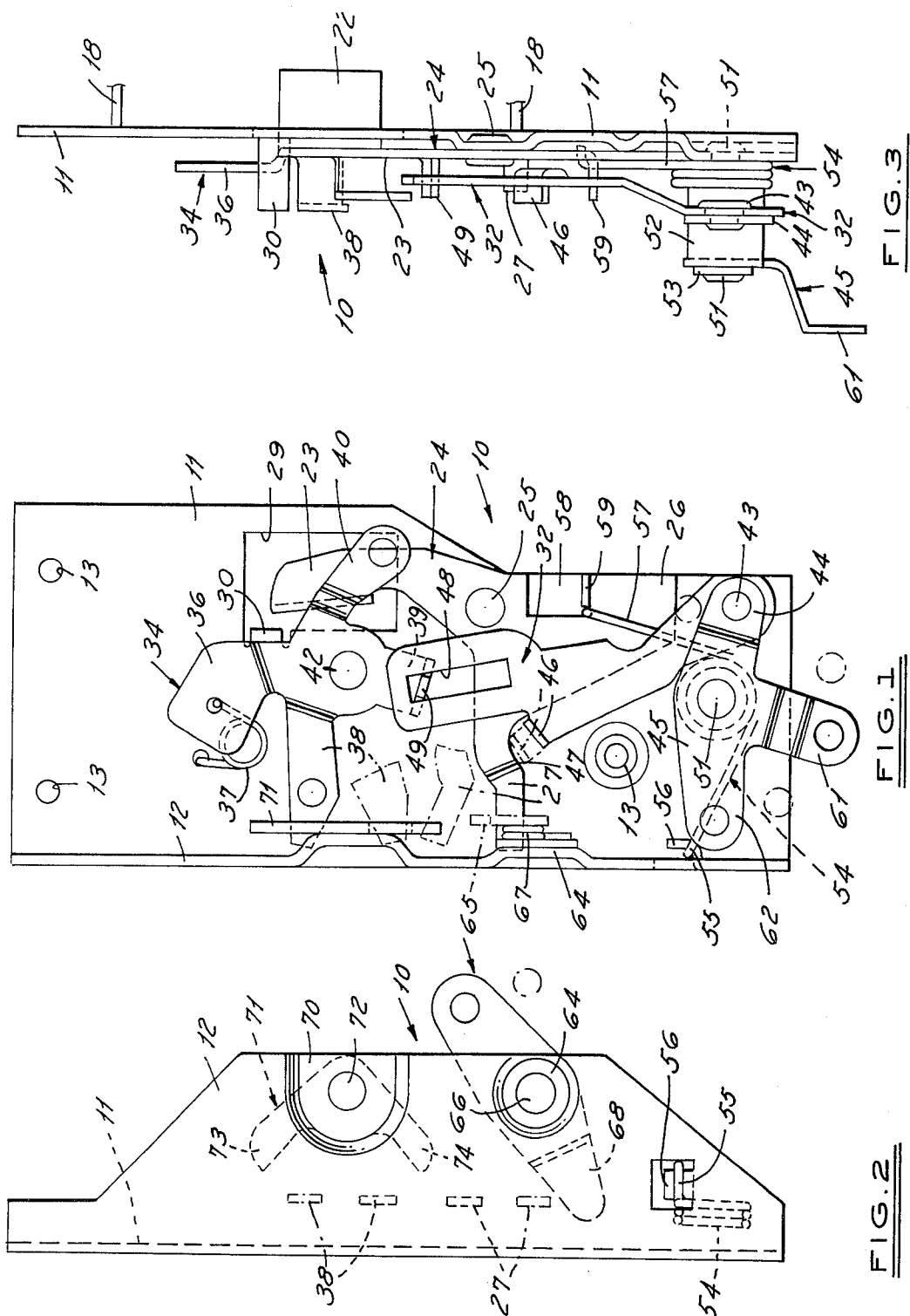
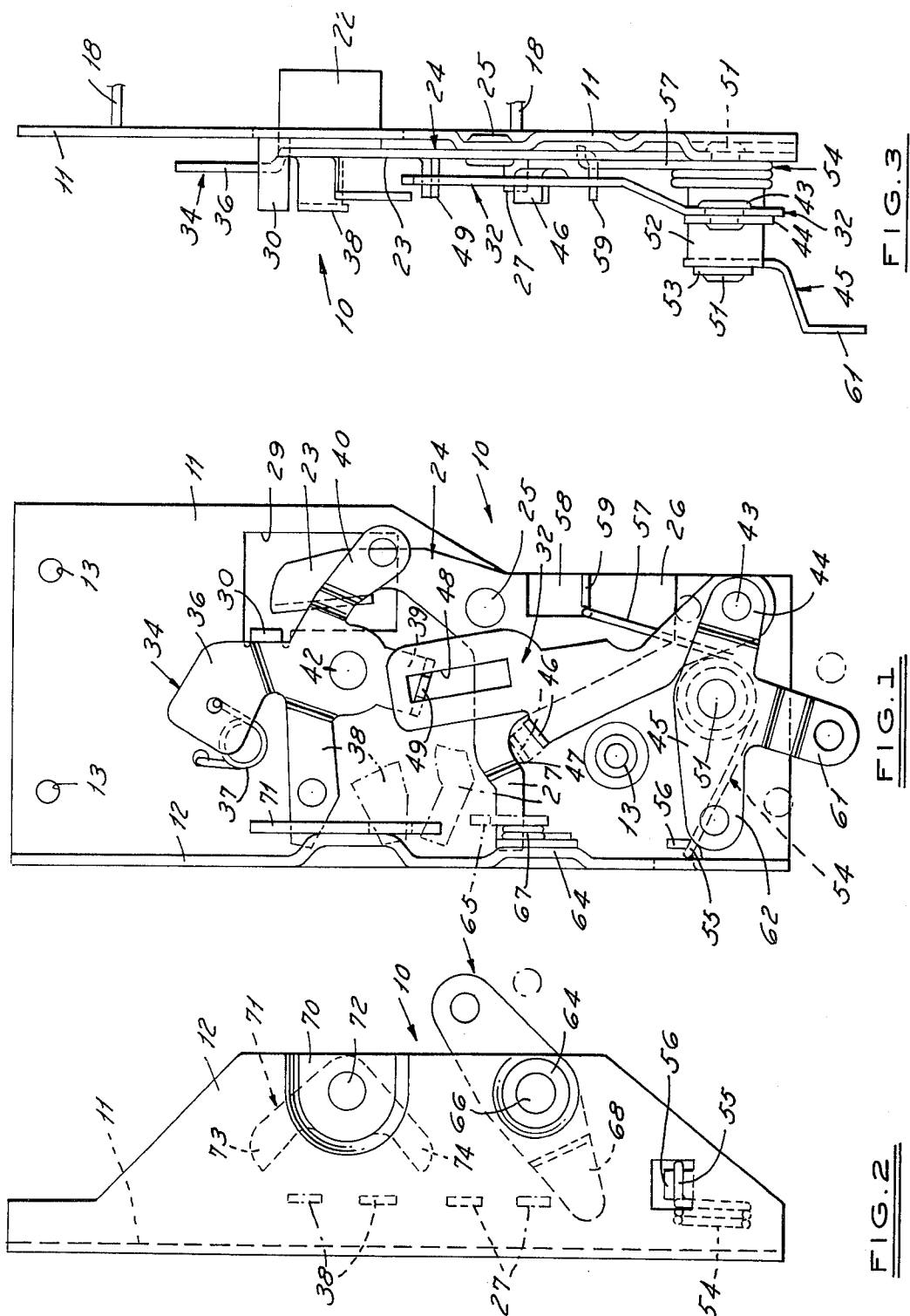


FIG. 4

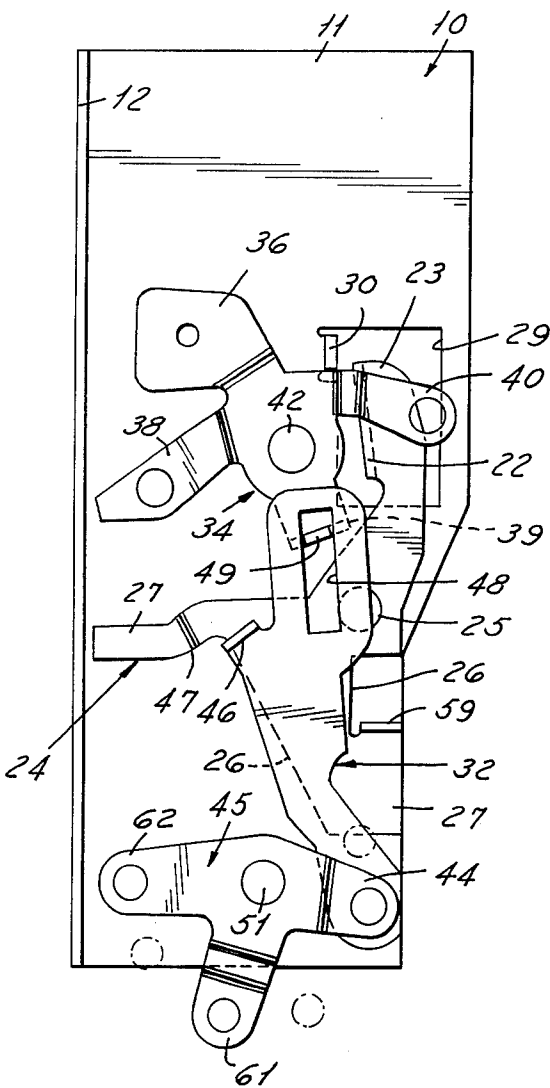


FIG. 5

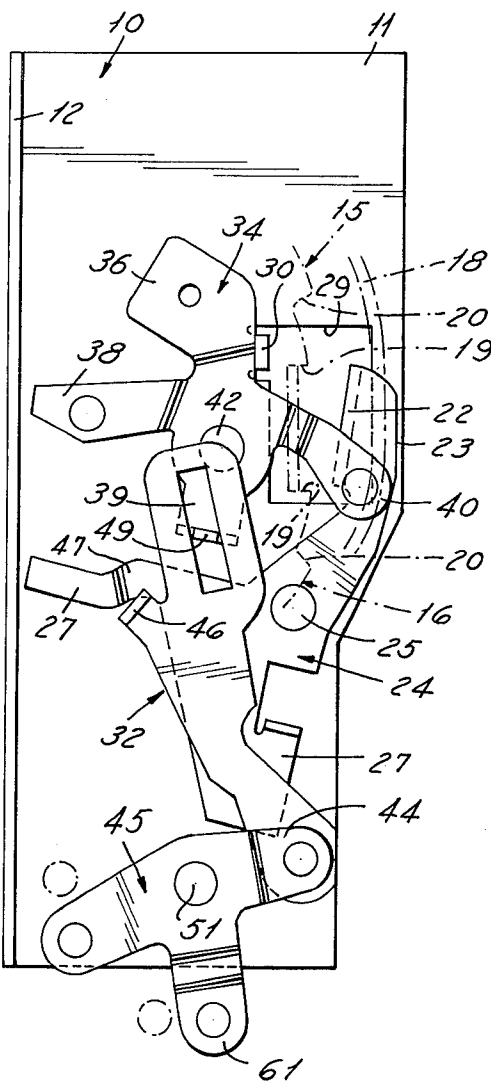


FIG. 6

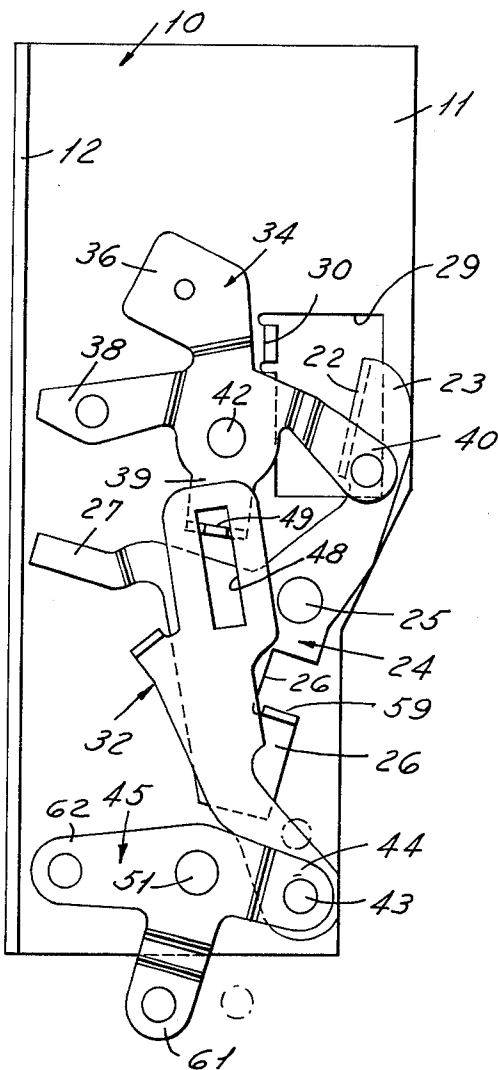
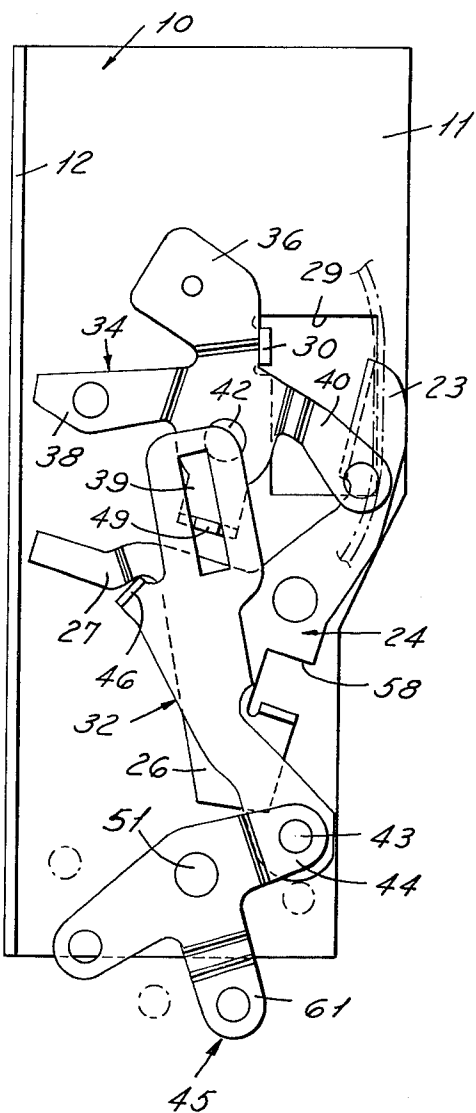


FIG. 7



AUTOMOTIVE SIDE DOOR LATCH

CROSS-REFERENCE TO RELATED APPLICATION

My co-pending application, Ser. No. 207,830, filed Dec. 14, 1971, now U.S. Pat. No. 3,784,241 of Jan. 8, 1974, discloses a generally similar type of latch or locking construction featuring forked dual jaw type overlapping latch members spring-biased toward a latching position, a locking lever quite similar to that of the present improvement, and a longitudinally shiftable, laterally or transversely cocked intermittent slide controlled by an outside operator. However, certain lost motion connections in that application between the latching pawl and the slide, and between the slide and the outside operator are not found in the lock herein disclosed, being replaced by simpler, fixed pivot arrangements, represented by rivets.

BACKGROUND OF THE INVENTION

1. Field of the Invention

The present lock, as fabricated in its entirety of inexpensive stamped parts operatively mounted by simple, inexpensive rivet-type pivot connections, is intended for installation on truck doors; and for that reason it forgoes relatively expensive refinements such as are commonly incorporated presently in passenger vehicle door locks.

2. Description of the Prior Art

The patent to Schlichter, U.S. Pat. No. 3,666,305 of May 30, 1972, is one of a number which disclose a double jaw-type automobile door lock, and is also generally related to the present improvement in respect to a tongue and slot type of freewheel connection between a locking lever and another latch component, but not to an actuator and disconnecter slide as in the present invention.

SUMMARY OF THE INVENTION

The improved lock is of an extremely simple construction, incorporating a minimum number of inexpensively stamped lever and slide parts; and the assembly of these parts, typically by rivet-type pivots, is an equally simple and inexpensive matter. The improved lock, insofar as its dual forked jaw and spring biasing means are concerned, presents all of the advantages of the construction of my above-identified application; and other arrangements like the locking lever of that application are carried forward herein without substantial change.

However, pursuant to the present improvement, the multiple arm pivotal blocking pawl of the lock is strongly biased by a torsion spring, which encircles the pivotal axis of the outside operator and anchors on a mounting plate of the lock and on the pawl, towards an automatic kick-off position, enabling the lock to be latched, with the pawl blocking the latch forks, when a conventional inside push button servicing the lock is depressed and the door is swung closed with the outside operator in an unlatching position. A further improvement in the lock resides in the fact that a longitudinal operator and disconnecter slide, as directly pivoted on an arm of the outside operated bell crank and shiftable upwards and downwards by the latter, also carries a laterally offset tab which either engages against an offset formation on a pawl arm to unlatch the pawl or, with the slide cocked somewhat about its outside operator pivot, to overrun the pawl arm offset, the unit thus

being in an overrunning or freewheeling locked condition. In this condition, an elongated slot in the slide enables the latter also to freewheel in relation to the locking lever.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a side elevational view showing the mounting of operating parts of the lock on a door-mounted plate stamping, positions of the components being depicted in solid line in an unlocked and latched condition of the lock, and in dotted line as they are shifted into positions in which the structure is unlocked and unlatched;

FIG. 2 is an end elevation of the lock as viewed from the left of FIG. 1, indicating a movement of a pawl arm from a solid line, latched position to a dotted line, unlatching position, as well as a movement of a locking lever arm from a solid line unlock position downward to a dotted line lock position;

FIG. 3 is an opposite end elevational view of the lock of the invention, as from the right of FIG. 1;

FIG. 4 is a side elevational view similar to FIG. 1, showing parts of the lock in a latched and locked condition of the latter, in which view (as in those of FIGS. 5, 6 and 7) certain spring-biased and other components appearing in FIG. 1 have been omitted for clarity;

FIG. 5 is a side elevation showing the lock in release position, i.e., unlocked and unlatched, with the dogging pawl appearing in solid line in its latch fork un-dogging position, and in dotted line in its dogging position;

FIG. 6 is a view similar to FIGS. 1, 4 and 5 of the lock in an automatic kick-off position; and

FIG. 7 is a similar side elevation depicting the lock in travel position as unlocked and unlatched.

DESCRIPTION OF A PREFERRED EMBODIMENT

The improved door lock of the invention, generally designated by the reference numeral 10, includes a sheet metal stamping having a main plate part 11 and a 90° flange affording a secondary plate part 12; and plate part 11 is fixedly secured, as at certain punch and boss formations 13 thereof, in a substantially flush relationship to and across the free, outer edge of the door, instanced above as a truck door.

As suggested in dot-dash line in FIG. 5, a pair of like latch forks 15, 16 are pivotally mounted on main mounting plate 11, being enclosed substantially in their entireties in an arcuate housing formation 18 projecting fixedly at 90° from the outer side of plate member 11. This arrangement of housing and dual latch forks itself constitutes no part of the present invention, but it is in any event more fully illustrated and described in my application identified above. In the manner described therein, the forks 15, 16 are operated against spring bias to releasably latch about a striker or keeper pin (not shown) conventionally affixed to the door frame. They are shown in FIG. 5 as being marginally shaped to provide full latch and safety latch shoulders 19, 20 respectively; and a single torsion spring (not shown) biases the forks 15, 16 respectively, clockwise and counterclockwise away from their latched position in which, per dotted line in FIG. 5, their latch shoulders 19 engage vertically and oppositely against a part of the pawl of the lock 10, thus to dog forks 15, 16 in a latched position surrounding the keeper pin of the lock, as fully described in my said co-pending application.

To this end, the latch fork shoulders 19 (and - in the case of a safety latch condition - the shoulders 20) are held in the door closed and latched position of FIGS. 1, 4 and 5 by a rectangular dogging pawl piece 22 integrally formed as a 90° offset of a rugged pawl arm 23, which arm is a first integral part of a 3-armed, bell crank-like pawl, generally designated 24. The latter is pivotally mounted by a rivet 25 on the side of mounting plate 11 opposite the side of forks 15, 16 at a point adjacent the junction of pawl arm 23 with the two other arms 26, 27 or pawl 24. These are respectively a generally horizontally extending, dedogging kick-off arm and a depending spring-operated arm. The dogging or blocking plate part 22 of pawl 24 extends laterally through a generous size rectangular opening 29 in mounting plate 11, thus permitting a liberal swing between the positions of part 22 appearing in solid and dotted line in FIG. 5. A lug 30 is struck and offset 90° from the plate to serve as a stop for a purpose to appear. Appropriate coil tension spring means (not shown) continually urges pawl 24 toward the fork-dogging position shown in FIGS. 1 and 4 and in dotted line in FIG. 5.

A generally vertically acting control, pawl-actuating and disconnecter slide 32 is mounted on the same side of plate 11 as the pawl 24, i.e., the side of the plate opposite that on which the latch fork structure 15, 16 is located. Slide 32 is an intermittent transmitter component of a latching and locking sub-assembly which further includes a multiple arm locking lever, generally designated 34.

In the general manner illustrated and described in my application, Ser. No. 207,830, said locking lever 34 is a stamping presenting (a) an upwardly projecting arm or ear 36 (FIG. 1) apertured to receive a conventional overcenter coil spring 37 which anchors on plate 11, the part 36 having provision (not shown) to operate locking lever 34 from the usual key-operated cylinder lock of the door; (b) a laterally projecting second arm 38 apertured to receive a connector (not shown) operated in the usual manner from a door molding-mounted push button; (c) a third depending arm 39 whose function is later detailed; and entirely as an option, a fourth angularly disposed arm 40, which will afford an operating connection as an alternative to the locking lever arm 38 in an installation of lock 10 in a high cab truck. As a feature in common with other simplified mounting provisions of the invention, the locking lever 34 is pivotally mounted on plate part 11 by a headed rivet 42.

The plural function actuator, disconnect and control slide 32 is a stamping which is considerably elongated in the vertical sense, being pivotally connected by a rivet 43 at its lower end to an offset arm 44 of a 3-arm outside operated bell crank lever 45 in a manner to be described. Medially of its length, the slide 32 carried a 90° offset pawl operating and dedogging tab or ear 46 which, upon upward movement of the slide from a solid line position of FIG. 1 to a dotted line position, engages beneath an integral offset formation 47 of laterally projecting pawl arm 27, thus to shift said arm to the dotted line position thereof in FIG. 1.

Slide 32 is provided adjacent its top with a considerably elongated slot 48; and this slot slidably receives, in a longitudinal lost motion manner, an ear or tang 49 offset 90° from the arm 39 of locking lever 34, as further shown and described in my co-pending application identified above.

The outside operated bell crank or lever 45 is pivotally mounted on plate 11 by a double headed rivet 51 which, as best shown in FIG. 3, extends through a cylindrical, barrel-like stud 52 backing against the mounting plate 11 and spaced at its opposite end from a head of rivet 51 by a washer 53. The outside operator 45 is fixedly connected medially to the outer end of stud 52; and (FIGS. 1 and 3) a strong coil torsion spring 54 medially surrounds and is centered by stud 52. One projecting arm 55 of spring 54 is offset to anchor on an ear 56 struck from the flanged plate part 12; and the opposite arm portion 57 of spring 54 is also offset to hook through a rectangular opening 58 stamped in the depending arm 26 of pawl 24, lying behind a struck-out ear 59 at that opening. Thus torsion spring 54 exerts a continuing strong force on pawl 24 to shift the latter clockwise about its plate pivot at 25, i.e., from the position of FIGS. 1 and 4 to the position of FIGS. 5, 6 and 7, in which the pawl is dedogged from between shoulders 19, 19 of FIG. 5. This occurs in the movement of pawl 24 in kicking off to latch releasing position, per FIGS. 5, 6 and 7.

An arm 61 of outside operated bell crank 45 is offset and apertured to receive, in a high-cab installation, an appropriate connector (not shown) operated from the usual handle of the door; and bell crank lever 45 also has a third arm 62 optionally connected, in a low-cab installation, to a similar handle or like operator.

As shown only in FIGS. 1 and 2, the 90° flange plate portion of the lock's mount has an inward boss at 64 at which an inside-operated remote control lever 65 of the lock pivotally mounts, again by a simple headed rivet 66, with a conventional snap-over spring 67 interposed between the plate part and lever and anchored to each to effect a desired completion of the swing of lever 65 in either rotative direction.

An outer end of lever 65 is apertured to receive an appropriate remote control connection; and an opposite inner end of said lever is laterally offset at 68 (FIG. 2) to effect engagement from beneath with the integral lateral kick-off arm 27 of pawl 24 in shifting said pawl at its said arm from the solid line, latched position of FIG. 2 to the upward unlatched position appearing in dotted line.

Furthermore, and as is best shown in FIG. 2, the plate flange 12 is stamped to provide a boss 70 at which a double armed auxiliary locking lever 71 is pivotally mounted by means of a rivet 72. An upper arm 73 of lever 71, as key operated from the exterior, swings downwardly into engagement with the lateral arm 38 of locking lever to swing the latter through said arm from the solid line, unlocked position of FIG. 2 to the dotted line locked position therein. Similarly, an integral lower arm 74 of auxiliary lever 71 is shiftable upwardly in engagement with arm 38 to restore locking lever 36 to the solid line locking position.

Commencing with the unlatching operation of lock 10 appearing in FIG. 1, a counterclockwise operation of outside operator 45 from its solid line position to its next, dot-dash depicted station (the lock being of course in unlocked position), causes slide 32 to shift upwardly and brings its offset ear or tab 46 into the dotted line position of FIG. 1 to engage and operate pawl arm 27 from its solid to its dotted line position. The corresponding setting appears in FIG. 7, it being noted that locking lever 34 is shown swung counterclockwise about its pivot rivet 42 to the unlocked position. Such

shift of slide 32 in the thus oriented position of the locking lever causes it to overrun at its slot 48 the tang 49 on locking lever arm 39 without interference by the latter because of the linear lost motion; and FIG. 7 shows lock 10 in its unlocked and unlatched travel condition.

FIG. 4 shows the condition of the lock in its dogged or latched and locked setting. It is placed in this setting from the position of its parts shown in FIG. 5 by a push button or key operated counterclockwise swing of locking lever 34, which causes the tang 49 on lock lever arm 39, in going to the right per FIG. 4 in engagement in the slide slot 48, to cam the slide to the right. This brings the slide's actuating ear or tab 46 to a position in which it will not, when the slide is operated upwardly, engage the offset 47 of pawl arm 27. Instead, it overruns or freewheels relative to said arm, disabling the lock for unlatching; and the idling shift of slide 34 is again accommodated at its slot 48. The lock remains in the dogging position of the pawl plate 22.

The unlocked and unlatched setting of the lock 10 represents the condition and position of the parts upon an undogging of the lock from the dotted line position of pawl plate 22 (per FIG. 5) to the solid line position of that figure, as also shown in FIG. 7. In this setting, a counterclockwise swing of outside operator 45 has caused slide 34 to shift upwardly, with the result that it has cammed the locking lever tang 49 from the position of FIG. 4 to that of FIG. 7, shifting the slide 32 from a substantially vertical orientation of its slot 48 to a position canted upwardly and to the left as viewed in FIG. 7. The same orientation can of course be obtained by a direct manipulation of locking lever 34. The slide tab or ear 46 is again brought directly beneath the offset 47 on pawl arm 27.

The kick-off position of FIG. 6 results when, after the door's garnish-mounted push button has been depressed the door is closed with its outside operating handle and the lock's lever 45 non-actuated. The heels of the forks 15, 16, in a spread condition thereof strike the keeper of the vehicle and close about it; and the strong main spring 54 of the lock overcomes the force of the tension spring (not shown) normally biasing pawl 24 toward its fork dogging position, swinging the pawl clockwise (FIG. 6) about its pivot at 25 to the undogging position. The kick-off is nullified by holding the outside operating handle in an actuated position as the door is closed with the push button depressed. In all of the lock positions appearing in FIGS. 1, 6 and 7, the upper arm 36 of lock lever 34 engages against the plate lug 30 as a stop.

What is claimed is:

1. In an automotive door lock, a fixed mounting plate, a pawl having a fixed pivotal mount to operate on said plate, said pawl having an operating arm, an operating lever similarly mounted by a fixed pivot on said plate, an intermittent operating slide having a fixed pivotal connection to said operating lever to be operated by the latter, a locking lever having a fixed pivot on said mounting plate, said slide having an actuator element engageable with the pawl operating arm to swing the pawl in a latch-releasing direction, the slide having a lost motion overrunning connection with a part of the locking lever and being engaged at said last-named connection by said locking lever part to shift the slide to a position in which its actuator element fails to engage said pawl operator arm, and spring means acting

directly on and between a fixed part of said mounting plate and a part fixed on said pawl, said spring means strongly biasing said pawl about its pivotal plate mount in a latch-releasing direction.

2. In an automotive door lock of the double fork latching type characterized by oppositely swingable latch forks releasably held in a door latching relation to one another, a fixed mounting plate on one side of which said forks operate, a pawl having a fixed pivotal mount to operate on the other side of said plate, said pawl being releasably engageable between said forks and having an operating arm, an operating lever similarly mounted by a fixed pivot on said other plate side, an elongated intermittent operating slide disposed on said other side and having a fixed pivotal connection to said operating lever to be operated by the latter in a longitudinal path paralleling the slide's elongation, a locking lever having a fixed pivot on said mounting plate to swing on said other side of the latter, said slide having an actuator element engageable with the pawl operating arm to swing the pawl in a fork-releasing direction, the slide having a lost motion, longitudinally overrunning connection with a part of the locking lever and being engaged at said last-named connection by said locking lever part to shift the slide to a position in which its actuator element fails to engage said pawl operator arm, and spring means acting directly on and between a fixed part of said mounting plate and a part fixed on said pawl, said spring means strongly biasing said pawl about its pivotal plate mount in a fork-releasing direction.

3. The door lock of claim 1, in which the fixed pivotings of said pawl, operating lever, slide and locking lever are all through the agency rivet-like components.

4. The door lock of claim 2, in which the fixed pivotings of said pawl, operating lever, slide and locking lever are all through the agency of simple rivet-like components.

5. The door lock of claim 1, in which said lost motion overrunning connection of said slide and locking lever part is at an opening in the slide which receives said lever part.

6. The door lock of claim 2, in which said lost motion overrunning connection of said slide and locking lever part is at an elongated slot in the slide which receives said lever part, the slot affording operating clearance for the slide relative to said locking lever part in the longitudinal overrunning of the slide.

7. The door lock of claim 1, in which said mounting plate has a 90° bent flange, said flange affording fixed pivotal connections for a pair of members which respectively operate said locking lever and said operating arm of the pawl.

8. The door lock of claim 2, in which said mounting plate has a 90° bent flange, said flange affording fixed pivotal connections for a pair of members which respectively operate said locking lever and said operating arm of the pawl.

9. The door lock of claim 1, in which said actuator element of the slide is an integral offset formation of the latter.

10. The door lock of claim 2, in which said actuator element of the slide is an integral offset formation of the latter.

11. In an automotive door lock, a fixed mounting plate, a pawl pivotally mounted on said plate, said pawl having an operating part, an operating lever pivotally

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mounted on said plate, an intermittent operating slide having a connection to said operating lever to be operated by the latter, said slide having an actuator element fixedly mounted thereon and engageable with the pawl operating part to swing the pawl in a latch-releasing direction, and spring means acting directly on and between a fixed part of said mounting plate and a part fixed on said pawl, said spring means strongly biasing said pawl about its pivotal plate mounted in a latch-releasing direction.

12. In an automotive door lock of the double fork latch type characterized by oppositely swingable latch forks releasably held in a door latching relation to one another, a fixed mounting plate on one side of which said forks operate, a pawl pivotally mounted on said plate, said pawl being releasably engageable between said forks and having an operating part, an operating lever pivotally mounted on said plate, an elongated in-

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termittent operating slide having a connection to said operating lever to be operated by the latter in a longitudinal path paralleling the slide's elongation, said slide having an actuator element engageable with the pawl operating part to swing the pawl in a fork releasing direction, and spring means acting directly on and between a fixed part of said mounting plate and a part fixed on said pawl, said spring means strongly biasing said pawl about its pivotal plate mount in a fork-releasing direction.

13. The door lock of claim 11, in which said actuator element of the slide is an integral offset formation of the latter.

14. The door lock of claim 12, in which said actuator element of the slide is an integral offset formation of the latter.

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