

Office de la Propriété Intellectuelle du Canada

Un organisme d'Industrie Canada Canadian Intellectual Property Office

An agency of Industry Canada

CA 2623963 C 2010/07/13

(11)(21) 2 623 963

(12) BREVET CANADIEN CANADIAN PATENT

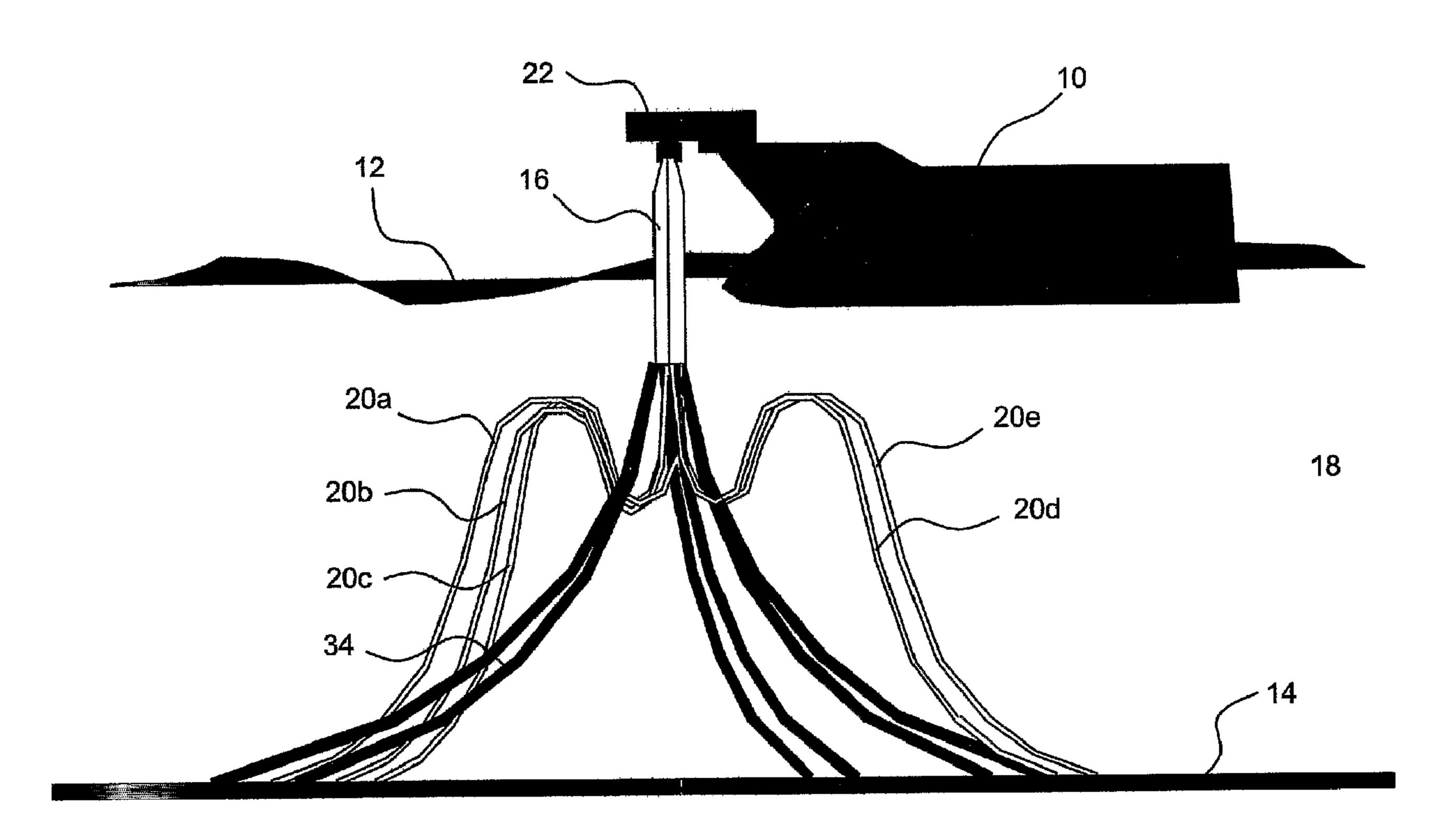
(13) **C**

- (86) Date de dépôt PCT/PCT Filing Date: 2005/09/30
- (87) Date publication PCT/PCT Publication Date: 2006/04/13
- (45) Date de délivrance/Issue Date: 2010/07/13
- (85) Entrée phase nationale/National Entry: 2008/03/27
- (86) N° demande PCT/PCT Application No.: GB 2005/003766
- (87) N° publication PCT/PCT Publication No.: 2006/037964
- (30) Priorité/Priority: 2004/10/01 (GB0421795.6)

- (51) Cl.Int./Int.Cl. *B63B 21/50* (2006.01), *B63B 22/02* (2006.01)
- (72) Inventeurs/Inventors:
 BAROSS, JOHN STEPHEN, GB;
 COLQUHOUN, ROBIN STUART, AE
- (73) Propriétaire/Owner: STANWELL CONSULTING LIMITED, GB
- (74) Agent: TEITELBAUM & MACLEAN

(54) Titre: ANCRAGE DE NAVIRE DE HAUTE MER ET SYSTEME DE CHARGEMENT DE TUBE GOULOTTE

(54) Title: OFFSHORE VESSEL MOORING AND RISER INBOARDING SYSTEM



(57) Abrégé/Abstract:

There is disclosed an offshore vessel mooring and riser inboarding system (12) for a vessel (10) such as an FPSO or FSO. In one embodiment, the system comprises a first mooring element (16) adapted to be located in an offshore environment (18); a riser (20) adapted to be coupled to the first mooring element; a connector assembly (22) 11 adapted to be mounted on the vessel, the connector assembly comprising a second mooring element (28); and a transfer line (32) adapted to be coupled to the riser; wherein the first and second mooring elements are adapted to be connected to facilitate coupling of the riser and the transfer line; and wherein the connector assembly is adapted to permit relative rotation between the vessel and the first mooring element about three mutually perpendicular. axes of rotation. This facilitates weathervaning of the vessel relative to the first mooring element, as well as pitch, roll, heave and surge, under applied wind, wave and/or tidal forces.





(12) INTERNATIONAL APPLICATION PUBLISHED UNDER THE PATENT COOPERATION TREATY (PCT)

(19) World Intellectual Property Organization

International Bureau





(43) International Publication Date 13 April 2006 (13.04.2006)

PCT

English

(10) International Publication Number WO 2006/037964 A1

(51) International Patent Classification: *B63B 21/50* (2006.01) *B63B 22/02* (2006.01)

(21) International Application Number:

PCT/GB2005/003766

(22) International Filing Date:

30 September 2005 (30.09.2005)

(25) Filing Language:

(26) Publication Language: English

(30) Priority Data:

0421795.6 1 October 2004 (01.10.2004) GB

(71) Applicant (for all designated States except US): STAN-WELL CONSULTING LIMITED [GB/GB]; The Old Brewery, Urquhart Road, Oldmeldrum, Aberdeenshire AB51 0EX (GB).

(72) Inventors; and

- (75) Inventors/Applicants (for US only): BAROSS, John, Stephen [GB/GB]; The Old Brewery, Urquhart Road, Oldmeldrum, Aberdeenshire AB51 0EX (GB). COLQUHOUN, Robin, Stuart [GB/AE]; Villa No. 7 41 Street, P.O. Box 2533, Sharjah (AE).
- (74) Agent: KENNEDYS PATENT AGENCY LIMITED; 185 St Vincent Street, Glasgow G2 5QD (GB).

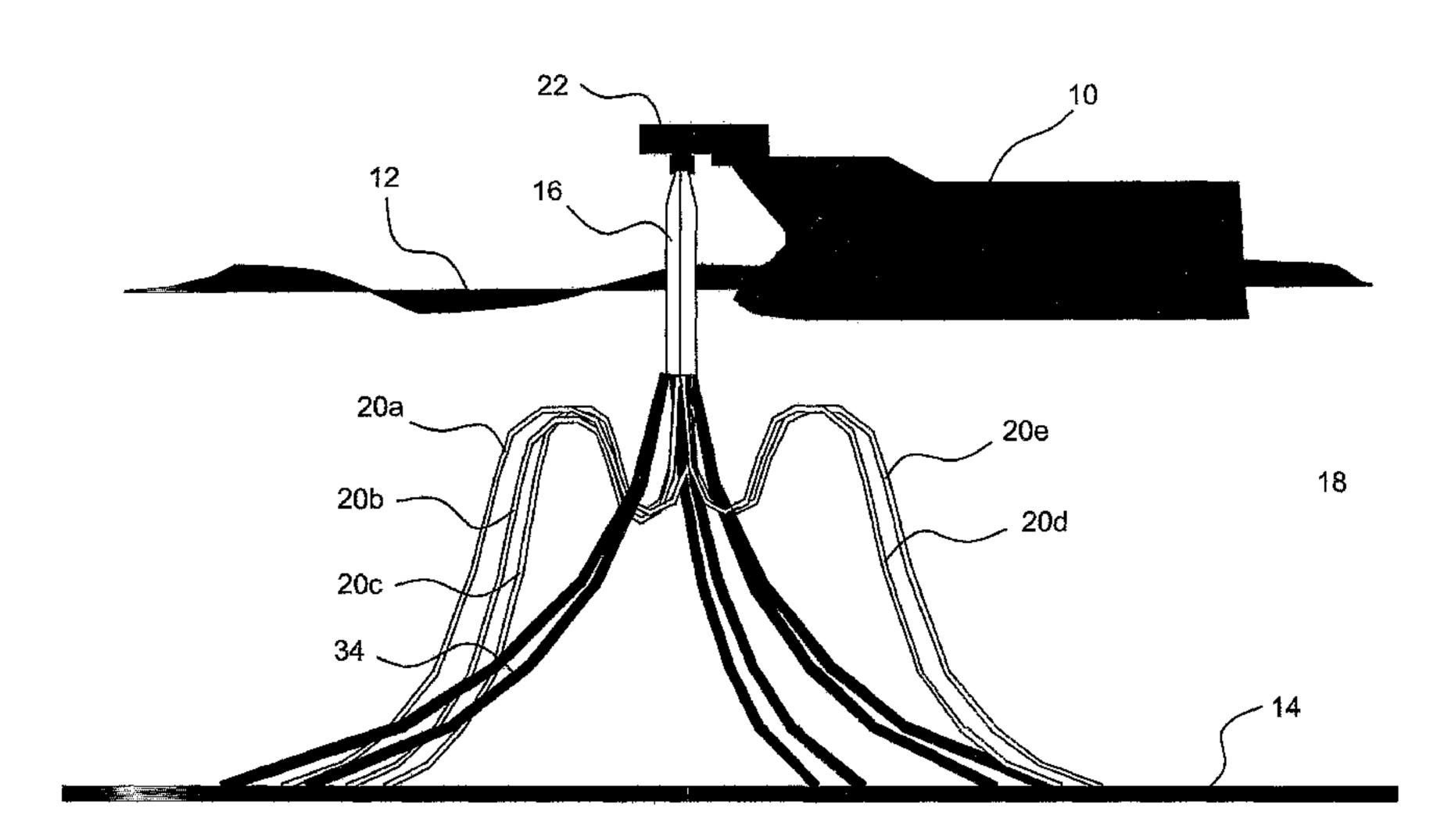
- (81) Designated States (unless otherwise indicated, for every kind of national protection available): AE, AG, AL, AM, AT, AU, AZ, BA, BB, BG, BR, BW, BY, BZ, CA, CH, CN, CO, CR, CU, CZ, DE, DK, DM, DZ, EC, EE, EG, ES, FI, GB, GD, GE, GH, GM, HR, HU, ID, IL, IN, IS, JP, KE, KG, KM, KP, KR, KZ, LC, LK, LR, LS, LT, LU, LV, LY, MA, MD, MG, MK, MN, MW, MX, MZ, NA, NG, NI, NO, NZ, OM, PG, PH, PL, PT, RO, RU, SC, SD, SE, SG, SK, SL, SM, SY, TJ, TM, TN, TR, TT, TZ, UA, UG, US, UZ, VC, VN, YU, ZA, ZM, ZW.
- (84) Designated States (unless otherwise indicated, for every kind of regional protection available): ARIPO (BW, GH, GM, KE, LS, MW, MZ, NA, SD, SL, SZ, TZ, UG, ZM, ZW), Eurasian (AM, AZ, BY, KG, KZ, MD, RU, TJ, TM), European (AT, BE, BG, CH, CY, CZ, DE, DK, EE, ES, FI, FR, GB, GR, HU, IE, IS, IT, LT, LU, LV, MC, NL, PL, PT, RO, SE, SI, SK, TR), OAPI (BF, BJ, CF, CG, CI, CM, GA, GN, GQ, GW, ML, MR, NE, SN, TD, TG).

Published:

- with international search report
- before the expiration of the time limit for amending the claims and to be republished in the event of receipt of amendments

For two-letter codes and other abbreviations, refer to the "Guidance Notes on Codes and Abbreviations" appearing at the beginning of each regular issue of the PCT Gazette.

(54) Title: OFFSHORE VESSEL MOORING AND RISER INBOARDING SYSTEM



(57) Abstract: There is disclosed an offshore vessel mooring and riser inboarding system (12) for a vessel (10) such as an FPSO or FSO. In one embodiment, the system comprises a first mooring element (16) adapted to be located in an offshore environment (18); a riser (20) adapted to be coupled to the first mooring element; a connector assembly (22) 11 adapted to be mounted on the vessel, the connector assembly comprising a second mooring element (28); and a transfer line (32) adapted to be coupled to the riser; wherein the first and second mooring elements are adapted to be connected to facilitate coupling of the riser and the transfer line; and wherein the connector assembly is adapted to permit relative rotation between the vessel and the first mooring element about three mutually perpendicular. axes of rotation. This facilitates weathervaning of the vessel relative to the first mooring element, as well as pitch, roll, heave and surge, under applied wind, wave and/or tidal forces.

PCT/GB2005/003766

1

Offshore	vessel	mooring	and	riser	inboarding	system

2

3 The present invention relates to an offshore vessel

- 4 mooring and riser inboarding system, and to a method of
- 5 mooring a vessel in an offshore environment. In
- 6 particular, but not exclusively, the present invention
- 7 relates to an offshore mooring and riser inboarding
- 8 system for a vessel such as a Floating Production Storage
- 9 and Offloading Vessel (FPSO) or a Floating Storage and
- 10 Offloading Vessel (FSO), and to a method of mooring a
- 11 vessel in an offshore environment.

- 13 In the oil and gas exploration and production industry,
- 14 well fluids (oil and gas) from offshore oil wells can be
- 15 transported to shore by submarine pipelines, laid on the
- 16 seabed. However, installing submarine pipelines involves
- 17 the use of dedicated pipelaying vessels, with a very
- 18 large associated capital expenditure, and the use of such
- 19 pipelines is therefore only commercially viable in
- limited circumstances. As a result, the exploitation of
- 21 oil and gas fields in certain areas, particularly those
- 22 far offshore or in deep water locations, has been shown

in the past to be of such marginal value that it has not been worth extracting the available oil and gas reserves. 3 To address this problem, there have been movements in the industry towards the exploitation of offshore oil and gas fields by the use of FPSOs or FSOs. An FPSO is moored in an offshore location and is typically coupled to a number of producing wells, for the temporary storage of produced 8 well fluids, which are periodically exported to shore by tankers. FPSOs typically include facilities for 10 separating recovered well fluids into different 11 constituents (oil, gas and water), so as to stabilise the 12 crude oil for onward transport by tanker. FSOs are 13 similarly moored and allow for the storage of recovered 14 well fluids, and may either be disconnected from their 15 moorings for travel to an offloading location, or the 16 recovered fluids may similarly be exported by tanker. In

contrast to FPSOs, however, FSOs do not have the facility 18 for separating the well fluids into different

constituents, and are therefore used in more limited 20

circumstances, typically for storage of stabilised, low 21

22 pressure crude.

23

19

Whilst some vessels are constructed and designed for 24

these purposes, many FPSOs and FSOs are conversions of 25

existing trading tankers. Converted vessels of this type 26

have usually functioned adequately, but there is a 27

continuing need for a substantial reduction in costs in 28

order to improve the economics of prospective development 29

and production of oil and gas fields, particularly those 30

which are currently deemed to be marginal. 31

Tankers used hitherto have often required extensive conversion work to enable them to operate as an FPSO or FSO. The extent of conversion work required depends upon factors including the particular circumstances under which the vessel is to be moored offshore. 7 A number of different systems have been developed for mooring vessels such as FPSOs and FSOs. For example, in one system, flowlines extend from the seabed to a mooring assembly which includes a buoyant mooring node, which is 10 located just below the sea surface. The node is moored 11 to the seabed by a number of mooring chains, and the 12 flowlines extend from the seabed to the node. A vessel 13 such as an FPSO is coupled to the node by a chafe chain 14 anchored on the vessel forecastle, and the chafe chain and the flowlines extend over a ramp on to the bow of the vessel. Whilst the FPSO can weathervane around the sea surface in the prevailing wind/tide, the degree of 18 movement permitted is limited (by the chafe chain and the 19 flowlines) to around one-and-a half rotations of the 20 vessel relative to the node in either rotational 21 direction; the vessel must then be either disconnected 22 and reset with the chain and flowlines in their original 23 positions, or rotated back to its median heading with the 24 aid of another vessel. Additional problems include that 25 the bow must be strengthened to accommodate loads 26 imparted by the chains and the flowlines, and that the 27 chain and the flowlines wear over time due to 28 scrubbing/chafing movement on the bow of the vessel. 29 30 In an alternative system, a buoyant canister is located 31

with a part above and a part below the sea surface. The 32

canister is moored to the seabed by a number of mooring 33

chains, which are connected to the canister, and the canister is connected to a vessel such as an FPSO by a cantilever frame on the FPSO. The frame is coupled to the canister by a swivel, to permit weathervaning of the vessel in the wind/tide, but is not free about the two orthogonal axes. In use, the canister requires to be maintained in a vertical orientation, to maintain connection with the frame and to permit weathervaning. Wind, wave and tidal loads on the FPSO are transmitted to the canister through the frame, and can be extremely 10 large. For example, in the event of a storm surge force 11 acting on the vessel tending to move the vessel astern, a 12 large bending moment is generated at the canister head. This is due to the distance between the location at which 14 the mooring chains are connected to the canister and the 15 location where the connecting frame is coupled to the 16 canister; this distance is dictated by a requirement to ensure that the FPSO does not strike the mooring lines. 18 As a result, the connecting frame experiences large 19 forces and is therefore a relatively heavy, bulky 20 structure, adding to the complexity of a tanker 21 conversion for use as an FPSO, and to the overa ${f 1}1$ weight 22 of the structure at the vessel bow. The canister 23 likewise has to be robust and heavy to sustain the large 24 bending moment. 25 26 Further systems involve the introduction of a rotating turret into the hull of a vessel, which permits

- 27
- 28
- engagement with a subsea buoy initially located below 29
- surface. Installation of systems of this type involves 30
- deep invasion into the structure of the vessel, 31
- necessitating a substantial period in drydock. 32
- systems are therefore relatively time-consuming and 33

5

costly to install. Furthermore, it is harder to achieve connection of the vessel to systems of this type, as the buoy must be below surface during approach of the vessel on station above it. 5 All of the systems developed to date have therefore suffered from a number of disadvantages, including: that they do not allow the vessel to weathervane continuously 8 without restriction; that they have been difficult to 9 install and hook up in the field; that they have had an 10 uncertain ability to allow the vessel to disconnect 11 rapidly, reliably and safely from the risers; and that 12 they have had a relatively restricted seastate 13 capability. Systems employing a chafe chain coupled to a 14 subsea node have also been prone to the risk of local 15 combined tension-bending fatigue in the upper mooring 16 chain where it traverses a ramp or fairlead on its route to a forecastle deck anchorage. 18 19 These problems apply in relation to the bringing inboard 20 of flow risers or lines (conduits for hydrocarbons or 21 other fluids), as well as to other risers or lines such 22 as power/control cables (for example, electrical lines 23 and hydraulic lines), and umbilicals. 24 25 It is amongst the objects of embodiments of the present 26 invention to obviate or mitigate at least one of the 27 foregoing disadvantages. 28

29

30

6

- 1 According to a first aspect of the present invention,
- 2 there is provided an offshore vessel mooring and riser
- 3 inboarding system, the system comprising:
- 4 a first mooring element adapted to be located in an
- 5 offshore environment;
- 6 a riser adapted to be coupled to the first mooring
- 7 element;
- 8 a connector assembly adapted to be mounted on a vessel,
- 9 the connector assembly comprising a second mooring
- 10 element; and
- 11 a transfer line adapted to be coupled to the riser;
- 12 wherein the first and second mooring elements are adapted
- 13 to be connected to facilitate coupling of the riser and
- 14 the transfer line;
- 15 and wherein the connector assembly is adapted to permit
- 16 relative rotation between the vessel and the first
- 17 mooring element about three mutually perpendicular axes
- 18 of rotation.

- 20 By permitting such relative rotation between the vessel
- 21 and the first mooring element, the present invention
- 22 facilitates movement of the vessel under external
- loading, in use, and reduces forces transmitted to/borne
- 24 by the vessel and the mooring and riser system
- 25 components. Accordingly, the connector assembly of the
- 26 present invention may not be required to support the
- 27 relatively large loads found in prior art systems. In
- 28 addition, the system permits all likely ranges of
- 29 movement of the vessel relative to the first mooring
- 30 element without excessive wear or damage to components
- 31 either of the system or to the vessel itself. In
- 32 particular, the vessel is able to weathervane (that is,
- 33 to move in response to applied wind, wave and/or tidal

7

1 loads, to face a direction of the prevailing wind, waves

- and/or tide), and to heave, pitch, roll, surge, sway and
- 3 yaw.

4

- 5 It will be understood that the three mutually
- 6 perpendicular axes of rotation may be taken about or with
- 7 reference to the first mooring element and may be taken
- 8 when the vessel is in a neutral or unloaded position.
- 9 Thus the first mooring element has three degrees of
- 10 freedom in its movement.

11

- 12 The riser may comprise or may take the form of a fluid
- 13 flow riser or flowline, which may be a conduit for
- 14 hydrocarbon containing fluids or other fluids.
- 15 Alternatively, the riser may comprise or may take the
- 16 form of a power and/or control cable, such as an
- 17 electrical and/or hydraulic cable. The riser may be an
- 18 umbilical comprising a flowline and one or more power
- 19 and/or control cable. The system may therefore permit
- 20 inboarding of any desired type of riser on to a vessel.
- 21 References herein to inboarding of a riser and to a riser
- inboarding system are to the bringing inboard or onboard
- 23 of a riser to a vessel and to such a system.

- 25 Where the riser comprises or takes the form of a fluid
- 26 flow riser or flowline, the transfer line may be a
- 27 transfer flowline, and connection of the first and second
- 28 mooring elements may facilitate flow of fluid between the
- 29 fluid flow riser, the transfer flowline and the vessel.
- 30 The transfer flowline may be for the passage of fluid
- 31 from the fluid flow riser into the transfer flowline and
- 32 to the vessel, or vice-versa.

8

1 Where the riser comprises or takes the form of a power

- 2 and/or control cable, the transfer line may provide an
- 3 electrical and/or hydraulic and/or other connection to
- 4 the riser. This may facilitate power supply, data
- 5 transmission and/or supply of hydraulic control fluid.

6

- 7 Preferably, the connector assembly further comprises a
- 8 support adapted to be mounted on the vessel, and the
- 9 second mooring element may be adapted to be mounted for
- 10 movement relative to the support. The support may be a
- 11 cantilever support and may be a support frame or the
- 12 like. The support may be located extending beyond a bow
- or stern of the vessel, or from the side of the vessel.
- 14 This may provide clearance for alignment and connection
- 15 of the first and second mooring elements.

16

- 17 Preferably also, the connector assembly further comprises
- an outer gimbal member, which may be mounted for rotation
- 19 relative to a part of the connector assembly, in
- 20 particular, the support. The assembly may also comprise
- 21 an inner gimbal member mounted for rotation relative to
- 22 the outer gimbal member. Additionally, the assembly may
- 23 comprise a rotatable coupling for facilitating rotation
- of the inner gimbal member relative to the first mooring
- 25 element. The rotatable coupling, inner gimbal member and
- 26 outer gimbal member together permit relative rotation
- 27 between the vessel and the first mooring element about
- 28 said axes of rotation.

- 30 The inner gimbal member may be rotatable about an inner
- 31 gimbal axis and the outer gimbal member about an outer

9

gimbal axis. The inner and outer gimbal member axes may be disposed substantially perpendicular to one another. This may facilitate relative rotation between the vessel and the first mooring element about two of the three mutually perpendicular axes of rotation. 5 The rotatable coupling may facilitate rotation between the inner gimbal member and the second mooring element, 8 to thereby permit relative rotation between the vessel 9 and the first mooring element about one of the three axes 10 of rotation. The rotatable coupling may therefore be 11 provided between the inner gimbal member and the second 12 mooring element. Alternatively, the rotatable coupling 13 may facilitate rotation between the second mooring 14 element and the first mooring element, to permit such 15 rotation. The rotatable coupling may thus be provided 16 between the first and second mooring elements and may be coupled to one of said elements. The rotatable coupling 18 may be a swivel and may comprise a rotary bearing, such 19 as a needle or roller bearing or a journal bearing of 20 special marine bearing material. 21 22 The inner and outer gimbal members may be annular rings 23 and the inner gimbal ring may be located within the outer 24 gimbal ring. In preferred embodiments, where the 25 connector assembly comprises a support adapted to be 26 mounted on the vessel, the outer gimbal member may be 27 rotatably mounted to the support and the inner gimbal 28 member may be rotatably mounted to the outer gimbal 29 member. Where the inner and outer gimbal members 30

comprise annular rings, the inner

gimbal ring may be mounted to the outer gimbal ring by inner trunnions and the outer gimbal ring may be mounted to the support by outer trunnions, the trunnions of the inner gimbal ring disposed perpendicular to those of the outer gimbal ring. 5 The connector assembly, in particular the support (which may be a cantilever structure), may be releasably mountable on the vessel. This may facilitate removal of the connector assembly if required. This may be desired, 1 O for example, where the connector assembly is provided on 11 a vessel such as a tanker converted for use as an FPSO or 12 FSO and it is desired to convert the vessel back for use 13 as a standard tanker. 14 15 Preferably, the first mooring element is buoyant and may 16 comprise or define a buoyant member. Alternatively, the system may comprise a separate buoyant member, and the 18 first mounting element may be coupled indirectly to the 19 buoyant member by a chain or the like. The first mooring 20 element or the buoyant member may be generally tubular, 21 and may optionally be a cylindrical tubular, and may 22 define an internal passage for receiving the main riser. 23 This may serve both to guide the riser into engagement with the first mooring element, and may also protect the riser from damage, for example, by contact with the 26 vessel in storm conditions. 27 28 The first mooring element and/or the buoyant member may be adapted to be located at surface prior to connection of the first and second mooring elements together.

- 29
- 30
- Accordingly, at least part of the first mooring element 32
- may protrude above a sea surface level. Alternatively, 33

11

1 the entire first mooring element may be adapted to be located below sea surface level. This may protect the first mooring element and the riser from loading, such as wind and wave loading. In this situation, the location of the first mooring element/buoyant member may be indicated by a marker buoy or the like. The first mooring element may be adapted to be moored to 8 or relative to a seabed in the offshore environment by a plurality of mooring lines. The mooring lines may be 10 catenary chains, mooring cables of wire or polymer rope 11 or other material, or a combination thereof. The mooring 12 lines may be adapted to bear loading of the vessel on the 13 first mooring element, to maintain the element on station and/or to prevent or minimise transmission of loads to 15 the riser. The mooring lines may be coupled to or adjacent to a lower end or portion of the first mooring element. This may provide sufficient clearance between 18 the mooring lines and the hull of the vessel, in use, 19 when the first and second mooring elements are connected. 20 21 In embodiments of the invention, the system may be a 22 mooring and riser inboarding system for a dynamically 23 positionable vessel. As is known in the industry, 24 dynamically positioned (DP) vessels are capable of 25 maintaining their geographical position through a control 26 system which includes a number of thrusters spaced around 27 the hull of the vessel. Where the system is designed for 28 use with such a vessel, it may not be necessary to moor 29 the first mooring element to or relative to the seabed, 30 as the mooring element does not require to maintain the 31 vessel on station. In these circumstances, the riser may 32

bear the relatively minor loading experienced by the

12

first mooring element due to, for example, wind, wave and tidal forces. 3 4 The first and second mooring elements may comprise or may define first and second connector elements, respectively, and may be adapted to be coupled together in a quick-7 connect and disconnect arrangement. This may facilitate alignment, connection and disconnection of the first and second connector elements, in use. One of the first and 9 second mooring elements may comprise a male member and 10 the other a female member, the female member adapted to receive the male member for engagement of the elements. 12 The connector assembly may comprise a locking arrangement for locking the first and second mooring elements together. The locking arrangement may comprise at least 15 one latch, locking dog or pin, which may be adapted to provide a releasable locking engagement between the first and second mooring elements. 18 19 The connector assembly may comprise an intermediate 20 connector for coupling the first and second mooring 21 elements together. The intermediate connector may be 22 secured to the first mooring element and thus may be 23 provided as part of the first mooring element, and may be 24 adapted to be releasably coupled to the second mooring 25 element. However, the intermediate connector may also be 26 releasably connected to the first mooring element. The 27 intermediate connector may also be adapted to support the 28 riser, and may define a riser hang-off unit. Releasably 29 securing the riser hang-off unit to the first mooring 30

element may facilitate access to the risers for

maintenance. The connector assembly may comprise a

31

13

jacking assembly or device, for selectively separating the first and second mooring elements by a desired or suitable distance. 4 Preferably, the system comprises a plurality of risers 5 and a corresponding plurality of transfer lines. transfer line may be associated with a corresponding riser. Alternatively, a single transfer line may be 8 associated with a plurality of risers. Where the riser is a fluid flow riser, each riser may be coupled to or 10 associated with a separate well, for the flow of well 11 fluids comprising oil and/or gas to the vessel. 12 13 The/each transfer line may be coupled to the/each 14 respective riser through a rotatable line coupling such 15 as a swivel or the like, which may be provided as part of or coupled to the second mooring element. This may facilitate weathervaning of the vessel whilst maintaining 18 connection between the riser and the transfer line. 19 20 Preferably, the connector assembly permits unlimited 21 rotation between the vessel and the first mooring element 22 about one of said axes of rotation, which may be a 23 vertical or Y-axis. This may facilitate full 24 weathervaning of the vessel around the first mooring 25 element. Rotation between the vessel and the first 26 mooring element about the other two of said axes of 27 rotation may be restricted depending upon dimensions of 28 the connector assembly, and in particular, by dimensions 29 of the inner and outer gimbal member. However, rotation 30 of at least up to 60 degrees from a neutral position 31 about the other two of said axes may be permitted, 32

providing up to 120 degrees total permissible rotation.

14

1 The system may comprise a device for adjusting a position

- 2 or orientation of the second mooring element relative to
- 3 the first mooring element, to facilitate connection of
- 4 the first and second mooring elements. In particular,
- 5 where the connector assembly comprises a rotatable
- 6 coupling and inner and outer gimbal members, the system
- 7 may comprise a device for adjusting a rotational position
- 8 of the outer gimbal member relative to the support;
- 9 and/or of the inner gimbal member relative to the outer
- 10 gimbal member; and/or a rotational orientation of the
- 11 first and second mooring elements.

12

- 13 The present invention may facilitate flow of well fluids
- 14 from a riser in the form of a fluid flowline through a
- 15 transfer flowline to a vessel. Additionally or
- 16 alternatively, the invention may be utilised in
- 17 circumstances where it is desired to offload fluid from
- 18 the vessel through the transfer flowline and into the
- 19 main flowline. This may facilitate discharge of fluid
- 20 carried by the vessel into a well, such as in order to
- 21 stimulate production, and/or to supply well fluids from
- 22 the vessel into a storage or transfer system, for
- 23 subsequent transfer to an alternative location.
- 24 References herein to transfer of fluid between the main
- 25 flowline, the transfer flowline and the vessel should
- 26 therefore be interpreted accordingly.

- 28 According to a second aspect of the present invention,
- 29 there is provided a method of mooring a vessel in an
- 30 offshore environment, the method comprising the steps of:
- 31 locating a first mooring element in an offshore
- 32 environment;
- 33 coupling a riser to the first mooring element;

15

- 1 connecting a second mooring element of a connector
- 2 assembly mounted on a vessel to the first mooring
- 3 element, such that relative rotation between the vessel
- 4 and the first mooring element about three mutually
- 5 perpendicular axes of rotation is permitted;
- 6 coupling a transfer line between the vessel and the
- 7 second mooring element; and
- 8 connecting the transfer line to the riser.

9

- 10 The method may comprise coupling a fluid flow riser to
- 11 the first mooring element, and coupling a transfer
- 12 flowline to the second mooring element. Following
- 13 connection of the transfer flowline to the fluid flow
- 14 riser, the method may comprise transferring fluid between
- 15 the fluid flow riser, the transfer flowline and the
- 16 vessel.

17

- 18 Further features of the method are defined above in
- 19 relation to the first aspect of the invention.

- 21 According to a third aspect of the present invention,
- 22 there is provided an offshore vessel mooring and riser
- inboarding system, the system comprising:
- 24 a first mooring element adapted to be located in an
- 25 offshore environment;
- 26 at least one riser adapted to be coupled to the first
- 27 mooring element;
- 28 a support adapted to be mounted on a vessel;
- an outer gimbal member mounted for rotation relative to
- 30 the support;
- an inner gimbal member mounted for rotation relative to
- 32 the outer gimbal member;

- 1 a second mooring element adapted for connection to the
- 2 first mooring element;
- a rotatable coupling for facilitating rotation of the
- 4 inner gimbal member relative to the first mooring
- 5 element; and
- 6 at least one transfer line adapted to be coupled between
- 7 the vessel and the second mooring element;
- 8 wherein, in use, the first and second mooring elements
- 9 are adapted to be connected to couple the transfer line
- 10 to the riser;
- 11 and wherein the rotatable coupling, the inner gimbal
- 12 member and the outer gimbal member together permit
- 13 rotation of the vessel relative to the first mooring
- 14 element.

15

- 16 There may be three degrees of freedom in movement of the
- 17 vessel relative to the first mooring element provided by
- 18 the inner and outer gimbal members and the rotatable
- 19 coupling.

20

- 21 According to a fourth aspect of the present invention,
- 22 there is provided a freely weathervaning bow or stern or
- 23 side mooring and riser inboarding system comprising:

24

- 25 means for mooring an offtake tanker or buffer tanker or
- 26 FPSO to the seabed and one or more fluid flowline and/or.
- 27 well control umbilical or electrical umbilical risers
- 28 connecting seabed facilities to the tanker or FPSO;

- 30 the mooring system comprising at least three chain or
- 31 rope or hybrid mooring lines with or without anchors,
- 32 each line being attached to padeyes at the lower end of a
- 33 cylindrical annular flotation canister, the upper end of

17

which is latched into a specially designed mooring swivel suspended within a gimbal in a structural cantilever projecting forward from the bow of the vessel at focsle deck level or at the stern or other position off the vessel's gunwale and being additionally supported by structural members springing from the vessel hull typically at focsle deck level or below; 8 the gimbal being so designed as to be capable of accommodating an angular deviation of the flotation 10 canister axis relative to the intersection of the 11 sagittal and transom planes of the ship of plus or minus 12 60 degrees in any direction arising as a result of the 13 first and second order motions of the ship subject only to the constraint of avoidance of interference with the bulbous bow; 16 each fluid flowline and umbilical running from the 18 direction of the seabed well or subsea facility and 19 ascending as a riser in the configuration of a Lazy Wave 20 or other suitable shape and entering the lower end of the 21 annular flotation canister through polymer bend-22 stiffeners attached to the lower end of the canister and 23 projecting below the canister and each flowline and 24 umbilical then ascending through the canister and through 25 the mooring swivel and gimbal to a hangoff frame above 26 and thence upwards via double-valved quick disconnects to 27 a multiple path swivel stack with its inner (geodetically 28 fixed azimuth) part standing on the upper part of the 29 quick disconnect assembly and riser hangoff unit within 30 the inner ring of the special mooring swivel and the 31 outer part of the multiple path swivel stack following the azimuth of the vessel (the swivel stack may consist 33

18

of one single path swivel alone in applications where there is only one fluid conduit riser and no umbilical); the fluid and electrical conduits from the outer part of the swivel stack passing down between the middle and 5 inner gimbal rings in the form of catenary jumpers 7 terminating at the vessel's pipework and cabling at a hangoff location on the stem of the vessel typically 8 between main deck and focsle deck level whence the fluid conduits proceed to Emergency Shutdown Valves (ESDs) and 10 a manifold inboard; 11 12 the multiple path swivel stack being shielded from the 13 weather within a protective housing mounted on the outer 14 ring of the mooring swivel so as to enable servicing and 15 maintenance work on the stack to be performed 16 conveniently and safely; 18 the riser hangoff frame being an integral part of a 19 specially designed Riser Hangoff Unit (RHU) incorporating 20 at its upper end the lower part of the multiple path 21 fluid conduit and electrical conduit quick disconnect 22 assembly (QDC) including the lower valve set and 23 incorporating at its lower end a specially designed 24 Latching Can (LC) containing the two sets of latches 25 which respectively lock the RHU into the flotation canister and lock the whole of the RHU-cum-flotationcanister assembly into the inner ring of the mooring 28 29 swivel; 30 the RHU being capable of being broken (unbolted) just 31 above the LC and the upper part of it together with QDC 32 and swivel stack being jacked up so as to provide access 33

19

above the LC for work in connection with initial pull-in and attachment of the risers and any subsequent changeout of the risers; 4 the vessel being able to abandon the mooring by activating the QDC and then releasing the flotation 7 canister with the RHU still locked into it and the buoyancy of the flotation canister being such as to ensure that the head of the canister and the RHU remain above water level all abandonment functions being 10 controlled remotely from the bridge of the vessel without any requirement for crew members to be present on or near the devices comprising the invention or the focsle area as a whole; 14 15 the mooring swivel incorporating a rotational indexing 16 motor or device to enable the inner part of the mooring swivel together with the QDC assembly and the inner part 18 of the multiple path swivel stack to be rotated to the 19 appropriate geodetic azimuth for recovery of the canister 20 and RHU regardless of the azimuth of the vessel; 21 22 a pair of winches being mounted in the cylindrical space 23 between the upper part of the QDC and the swivel stack 24 with the winch lines running down through the QDC for 25 attachment to the head of the RHU on the floating 26 canister (by crew standing on the structure hanging from 27 the inner ring of the mooring swivel) as the vessel approaches for pickup and reconnection so that the 29 canister can then be pulled towards the vessel and the 30 vessel towards the canister with the gimbal automatically 31 coming into appropriate alignment for mating and 32 latching; 33

1	
2	the hydraulic supply to the QDC and to the latches in the
3	LC being routed from the vessel via fluid path swivels in
4	the swivel stack and the locks and hydraulic circuitry
5	and controls being designed so as to provide appropriate
6	functional interlocks and fail-safe behaviour.
7	
8	In a further aspect of the present invention, there is
9	provided a connector assembly as defined in the attached
0	claims. Further features of the connector assembly are
1	defined above.
.2	
.3	Embodiments of the present invention will now be
4	described, by way of example only, with reference to the
.5	accompanying drawings, in which:
.6	
.7	Fig 1 is a schematic side view of a vessel shown
.8	moored to an offshore mooring and riser inboarding
.9	system in accordance with a preferred embodiment of
0	the present invention;
21	
22	Fig 2 is an enlarged, perspective view of the system
23	and a bow of the vessel shown in Fig 1;
24	
25	Fig 3 is an enlarged, partial cross-sectional view
26	of part of a first mooring element, and part of a
27	connector assembly comprising a second mooring
28	element, of the system shown in Fig 1;
29	
3 0	Fig 4 is a view of the complete first mooring
31	element of the system shown in Fig 1;
32	

1	Fig 5 is a cross-sectional view of the first mooring
2	element taking about the line A-A of Fig 4;
3	
4	Fig 6 is an enlarged view of part of the system
5	shown in Fig 1, taken from the other side, and
6	illustrated when the vessel experiences a large
7	surge force in an astern direction;
8	
9	Fig 7 is an enlarged front view of part of the
0	system shown in Fig 1, illustrated when the vessel
1	experiences a large force in a thwartship direction;
2	
3	Fig 8 is an enlarged view of part of a locking
4	assembly and a riser take-off unit of the system
5	shown in Fig 1;
6	
7	Fig 9 is a schematic cross-sectional view of the
.8	first mooring element of the system shown in Fig 1,
9 -	taken at a location where it abuts a riser hang-off
0	of the system;
1	
2	Figs 10 to 13 are views illustrating the steps in a
3	method of connecting the first and second mooring
4	elements of the system shown in Fig 1 together;
5	
	Fig 14 is an enlarged view of a bottom part of the
27	first mooring element shown in Fig 4;
8	· · · · · · · · · · · · · · · · · · ·
29	Fig 15 is a view illustrating part of the system of
3 0	Fig 1 during riser installation, changeout, or
3 1	inspection and maintenance;

1	Fig 16 is a view illustrating part of the system
2	during a maintenance procedure;
3	
4	Fig 17 is a perspective view of a vessel shown
5	moored to an offshore mooring and flowline system in
6	accordance with an alternative embodiment of the
7	present invention;
8	
9	Fig 18 is a side view of a bow of a vessel shown
L O	moored to an offshore mooring and flowline system in
11	accordance with a further alternative embodiment of
12	the present invention;
13	
14	Fig 19 is a view of the system of Fig 18 prior to
15	connection of first and second mooring elements of
16	the system together or after disconnection;
17	
18	Fig 20 is a side view of a bow of a vessel shown
19	moored to an offshore mooring and flowline system in
20	accordance with a still further alternative
21	embodiment of the present invention; and
22	
23	Fig 21 is a view of the system of Fig 20 prior to
24	connection of first and second mooring elements of
25	the system together.
26	
27	Turning firstly to Fig 1, there is shown a schematic side
28	view of a vessel 10, the vessel 10 shown moored to an
29	offshore mooring and riser inboarding system in
30	accordance with a preferred embodiment of the present
31	invention, the system indicated generally by reference
32	numeral 12. The system 12 is shown in more detail in the
33	enlarged, perspective view of Fig 2 and in Fig 3, which

is an enlarged, partial cross-sectional view of part of

2 the system 12 shown in Fig 1.

- 4 The vessel 10 may take the form of an FPSO, FSO, an
- off-take tanker or a buffer tanker, and is shown in the
- 6 figures moored to a seabed 14 by the system 12, for the
- 7 transfer of well fluids such as oil or gas to the vessel
- 8 10. The system 12 comprises a first mooring element in
- 9 the form of a flotation canister 16, which is shown
- 10 separately in Fig 4 and in the cross-sectional view of
- 11 Fig 5, which is taken about line A-A of Fig 4. As shown
- in Fig 1, the flotation canister 16 is located in an
- offshore environment 18, such as a sea or ocean. The
- 14 system 12 also comprises at least one and, in the
- illustrated, preferred embodiment, a number of risers,
- 16 five of which are shown in Fig 1 and given the reference
- numerals 20a to 20e. The risers 20a to 20e take the form
- 18 of fluid flow risers or flowlines and extend from the
- 19 seabed 14 into the flotation canister 16. The inherent
- 20 buoyancy of the main fluid flowlines 20a to 20e is
- 1 utilised to arrange the lines in a "lazy wave"
- 22 configuration, which reduces loading on the flowlines and
- 23 allows for movement of the flotation canister 16 without
- 24 transferring excessive loading on to the flow lines 20a
- to 20e. However, the canister 16 includes buoyancy
- 26 chambers 17 and is thus inherently buoyant, to support
- 27 the risers 20. It will be understood that any other
- alternative configuration of the flowlines 20a to 20e may
- 29 be employed. Each of the main fluid flowlines 20a to 20e
- 30 extend from a respective subsea wellhead (not shown) or
- 31 pumping facility provided on the seabed 14 (not shown),
- 32 for supplying well fluids through the respective main
- 33 flowline 20 to the vessel 10.

The system also comprises a connector assembly 22 which includes a support in the form of a frame 24 which is mounted on a bow 26 of the vessel 10 on the forecastle 27 as best shown in Fig 2. The connector assembly includes a second mooring element of the system 12, which is indicated generally by reference numeral 28. The second mooring element 28 forms a second connector for coupling to a first connector defined by a neck 30 of the flotation canister 16. 10 11 The system 12 also comprises at least one and, in the 12 illustrated, preferred embodiment, a number of transfer 13 lines, six of which are shown and given the reference 14 numerals 32a to 32e, each of which corresponds to a respective riser 20. The transfer flowlines are provided 16 as catenary jumpers 32a to 32e, and are each coupled between the vessel 10 and the second connector 28, and 18 serve for transfer of fluid through the respective riser 19 20 to the vessel 10 when the second connector 28 is 20 coupled to the flotation canister 16, as will be 21 described in more detail below. 22 23 The flotation canister 16 is moored in the offshore 24 environment 18 by a number of mooring lines 34, which are 25 coupled to padeyes on the canister 16. As shown in Fig 26 2, there may be three such mooring lines 34a to 34c and the mooring lines may be catenary chains, cables, wires - 28 or a combination thereof. As will be understood by 29 persons skilled in the art, selection of the appropriate 30 mooring line 34 depends upon factors including water 31 depth in the offshore environment 18. In the illustrated 32 embodiment, however, catenary chains 34a to 34c are 33

25

employed, which are anchored to the seabed 14 and serve for maintaining position of the flotation can ister 16 within accepted tolerances, and for supporting loading transmitted to the canister 16 by the vessel 10, in use. 5 As will be described in more detail below, the connector assembly 22 permits a relative rotation between the vessel 10 and the flotation canister 16 about three mutually perpendicular axes of rotation X, Y and Z, as shown in Fig 2. The axes X and Z are in a horizontal 10 plane and are perpendicular to one another. The Y axis is in a vertical plane and is perpendicular to both the X 12 and Z axes. In a neutral position of the system 12, 13 where the flotation canister 16 is vertically oriented 14 and assuming no external loading on the vessel 10, the X 15 axis is parallel to a main, longitudinal or sagittal axis 16 of the vessel 10; the Y axis is parallel to a main, longitudinal axis of the flotation canister ${f 1}$ 6; and the Z 18 axis is parallel to a transom or transverse plane of the 19 vessel 10. 20 21 By this arrangement, the vessel 10 may weathervane 22 according to the prevailing wind, wave and/or tide where 23 the vessel is turned to face the direction of applied 24 loading, by rotation about the Y axis. Additionally, the 25 connector assembly 22 permits an angular deviation 26 between the vessel 10 and the flotation canister 16 of up 27

to 60 degrees astern and 15 degrees forward from the

astern direction. It will be noted that certain

29

30

31

32

33

neutral position of Fig 2 about the Z axis, as shown in

Fig 6, which is an enlarged view of the system 12 shown

when the vessel 10 experiences a large surge force in an

components of the system 12 have been omitted from Fig 6,

for ease of illustration. Relative rotation between the vessel 10 and the flotation canister 16 about the X axis is shown in Fig 7, where the vessel 10 is experiencing a large thwartship force derived from the combination of, for example, low frequency yaw and sway and wave 5 frequency roll. The relative dimensions of the system 12 and in particular of the connector assembly 22 are such that unlimited rotation of the vessel 10 in a path around a circumference of the flotation canister 16 is possible (about the Y axis). Additionally, these dimensions are 10 such that an angular misalignment of up to 60 degrees 11 from the vertical is possible in any other direction, as 12 shown in Figs 6 and 7, subject only to the constraint of 13 avoiding interference with the bulbous bow. Thus a total 14 relative movement of up to about 75 degrees about the Z 15 axis is possible (60 degrees during surge astern and 16 about 15 degrees during surge forward) and of up to 120 degrees about the X axis. The canister 16 includes 18 bumper strips 21 which prevent damage to the canister 19 through accidental contact with the vessel bow 26. 20 21 The system 12 therefore facilitates vessel mooring and 22 riser inboarding even where the vessel experiences 23 extremes of loading due to wind, wave and/or tidal 24 forces. 25 26 The structure and method of operation of the system 12 27 will now be described in more detail, with reference also 28 to Figs 8 to 17. 29 30

31 As best shown in Figs 2 and 3, the support frame 24

32 includes outer support arms 36 and 38 by which the second

33 connector 28 is suspended from the vessel 10. The

1 connector assembly 22 includes an outer gimbal member in

- 2 the form of an outer gimbal ring 40, which is rotatably
- 3 mounted between the outer support arms 36 and 38 by
- 4 trunnions 42. The connector assembly 22 also includes an
- 5 inner gimbal member in the form of an inner gimbal ring
- 6 44, which is rotatably mounted to the outer gimbal ring
- 7 40 by trunnions 46, which are best shown in Fig 6. The
- 8 trunnions 42 and 46 are disposed on axes which are
- 9 perpendicular to one another, such that respective axes
- 10 of rotation of the outer and inner gimbal rings 40 and 44
- 11 are also perpendicular.

12

- 13 An inner flanged swivel ring 48 is mounted and suspended
- 14 from the inner gimbal ring 44, and the inner gimbal ring
- 15 44 and inner swivel ring 48 together define a swivel 50.
- 16 This facilitates rotation between the inner gimbal ring
- 17 44 and the inner swivel ring 48, via suitable bearings
- 18 (not shown). An integral structure in the form of a
- 19 lower housing 52 is coupled to and extends downwardly
- from the inner swivel ring 48, and the second connector
- 21 28 is coupled to the inner swivel ring 48 and extends
- along the lower housing 52 and is thus suspended from the
- 23 inner gimbal ring 44.

- The outer gimbal ring 40 facilitates angular displacement
- 26 between the vessel 10 and the flotation canister 16 in
- the fore and aft directions, as illustrated in Fig 6, by
- rotation about the outer support arms 36 and 38 on the
- 29 trunnions 42. In a similar fashion, the inner gimbal
- 30 ring 44 permits annular displacement between the vessel
- 31 10 and the flotation canister 16 in the thwartship
- 32 direction of Fig 7, by rotation of the inner gimbal ring

1 44 relative to the outer gimbal ring 40 on the trunnions

2 46.

3

- 4 The second connector 28 includes a housing 54 which is
- 5 located within and secured relative to the inner swivel
- fring 48. The second connector 28 includes a locking
- 7 mechanism 56 which forms an upper part of a quick
- 8 disconnect (QDC) 58, which is also shown in Fig 8. A
- 9 lower part 63 of the QDC 58 forms part of a riser hang
- 10 off unit (RHU) 60, which also includes a latching can 61
- 11 that is secured to the canister neck 30 by latches 62a.
- 12 The RHU 60 supports the risers 20, which extend upwardly
- through a central shaft 64 of the canister 16, and
- 14 includes a latching can. The RHU 60 is normally
- permanently latched into the head or neck 30 of the
- 16 canister 16 and constitutes an integral part of the
- 17 canister.

18

- 19 Fig 9 illustrates flow risers 20a to 20f in cross-section
- 20 at the interface between the canister 16 and the QDC 58.
- 21 Fig 9 also illustrates hydraulic and electrical umbilical
- 22 cores 66 and shows QDC valve and latch actuator hydraulic
- cores 68, which are used to control operation of the QDC
- 24 58.

- 26 As shown in Fig 8, the housing 54 of the second connector
- 27 28 carries a multiple path swivel stack 70, which
- includes a number of primary fluid swivels 72a to 72f,
- each associated with a respective riser 20 and jumper 32.
- 30 The primary fluid swivels 72 provide fluid connection
- 31 between a riser 20 and the respective jumper 32, and
- 32 facilitates unlimited rotation of the vessel 10 about the
- 33 canister 16 whilst maintaining fluid flow. Connectors

29

1 may extend between the swivels 72 and the risers 20. A

- 2 secondary swivel assembly 74 is provided above or below
- 3 the primary fluid swivels 72, and provides for canister
- 4 to mooring swivel latch actuation; QDC valve actuation;
- 5 QDC release actuation; umbilical hydraulic line
- 6 connection; hydraulic core 68 connection; and connection
- 7 to other ancillary equipment. An optional methanol line
- 8 76 and electrical slipring box 78, for handling the
- 9 umbilical power and signal cores 68, is also shown in Fig
- 10 8. The housing 54 contains piping extending from the QDC
- 11 58 to the swivel stack 70 and pull-in winches (not
- 12 shown), which are used during connection, as will be
- 13 described below.

- 15 Turning now to Figs 10 to 13, the method of connection of
- 16 the second connector 28 to the flotation canister 16 will
- 17 now be described. In Fig 10, the vessel 10 is shown
- approaching the canister 16, which is shown with the RHU
- 19 60 latched to the canister neck 30 by latches 62b. A
- 20 protective cover 80 is also shown in place on the RHU 60.
- 21 A connector line 82 is coupled to the cap 80 and is
- 22 marked by a buoy 84. When it is desired to mate the
- 23 second connector 28 with the flotation canister 16, a
- 24 winch line 86 is hooked on to the connector line 82, as
- shown in Fig 10. The connector line 82 is then reeled-
- 26 in, as shown in Fig 11, and bears against a lower end of
- the lower housing 52, rotating the connector assembly 22
- about the support arms 36 and 38 by the outer gimbal ring
- 29 40. Automatic alignment of the swivel 50 and the
- 30 canister head of the RHU 60 is assured during pull-in by
- the two angular degrees of freedom of the gimbals 40, 44
- 32 and two degrees of freedom of the canister 16.

30

When the canister 16 is picked up, it is important that the azimuth of the riser array and lower part of the QDC assembly 58 around the central axis of the stack match with the azimuth of riser connections on the underside of the upper part of the QDC assembly 58. Final adjustment can be achieved with the aid of simple mechanical guides(not shown), but the azimuths must first be brought into approximate alignment using an indexing system (not 8 shown). This is done by fitting a gear ring in the 9 around the stack at a convenient level, such as in the 10 swivel 50, with an associated hydraulic motor and 11 gearbox. An operator with a remote (wandering lead) 12 control box stands in a position where he can observe the 13 RHU 60 and canister 16 approaching and turns the stack so 14 as to match the azimuths of the upper and lower parts. 15 16 Accordingly, the second connector 28 is rotated to align it with the RHU 60, by rotating the swivel 50 the 18 indexing system. Further reeling-in then draws the RHU 19 60 into an internal passage 88 defined by the lower 20 housing 52, as shown in Fig 12, and the vessel 10 is then 21 moved forwards to position on station with the canister 22 16 in a vertical orientation, as shown in Fig 13. The 23 canister 16 is supported and the cap 80 removed, 24 following which the canister 16 is drawn up and the 25 locking mechanism 56 is operated to engage an upper ring 26 90 of the RHU, as shown in Fig 3. The lower latches 62a 27 are also actuated to engage the lower housing 52, and the 28 canister 16 is locked and supported 16 within the housing 29 28 and is ready for operation. 30

- 32 Following connection and appropriate testing of integrity
- 33 of the system 12, fluid communication between the risers

20 and the vessel 10, through the primary fluid swivels 72 and jumpers 32, may commence. The outer gimbal ring 40, inner gimbal ring 44 and swivel 50 permit a full range of motion of the vessel under wind, wave and tidal loading, including any combination of pitch, heave, roll, 5 surge, sway and yaw and also weathervaning (a particular manifestation of yaw), without requiring disconnect from the flotation canister 16. Movement of the canister 16 under load, as illustrated for example in Figs 6 and 7, 9 causes a degree of flexing in the risers 20 where they 10 enter the canister 16. Accordingly, as shown in Fig 14, 11 which is an enlarged view of a lower part of the 12 flotation canister 16, bend stiffeners 92 are provided 13 around the risers 20; two such bend stiffeners 92a and 14 92b are shown on the risers 20a and 20b. These provide protection for the risers 20 against damage through 16 contact with the canister 16. 18 When it is desired to abandon connection with the 19 flotation canister 16, a controlled abandonment may be 20 carried out in fair weather. This is achieved by 21 releasing the locking mechanism 56 and the latches 62a 22 and lowering the canister 16 to the position of Fig 13. 23 This provides a space 94 facilitating access to re-secure 24 the protective cover 80 and connector line. The RHU 60 is then lowered out of the lower housing 52. The 26 connector line 86 can then be disconnected and the vessel 10 may move away from the location of the canister 16, 28 for example, for passage to discharge location or if it 29 is desired to abandon the oil/gas field. However, in 30 certain circumstances, such as in an emergency 31

abandonment or in a heavy seaway, no crew are allowed in

the vicinity and the RHU 60 may be released without a protective cover.

3

- 1 Fig 15 illustrates an optional maintenance procedure,
- 5 where the locking mechanism 56 is released and a jack
- 6 assembly 89 actuated. This carries the housing 54
- 7 upwardly, to provide a space 94 for access to the RHU 60.

8

- 9 In other circumstances, it may be desired or required to
- 10 access the RHU 60, to carry out maintenance work, such as
- 11 on supports for the risers 20 or to carry out riser
- installation/changeout. To enable this, the latch
- elements 62a are operated to release a lower ring 96 of
- 14 the RHU 60, and the jack assembly 89 is actuated to carry
- 15 the second connector housing 54 and the RHU 60 upwardly,
- 16 to provide a space 98 for access to the inside of the RHU
- 17 60 and the risers 20, as shown in Fig 16.

- 19 Indeed, Fig 16 also illustrates first connection of the
- 20 FPSO 10 to the canister 16; the canister 16 (without the
- 21 RHU 60) and its moorings 34 are installed before the FPSO
- 22 10 arrives at the field. The risers 20 are likewise
- installed before FPSO 10 arrival and are buoyed off.
- 24 The RHU is installed on the FPSO 10 at the dockyard.
- 25 The upper part of the RHU 60 is the lower part of the QDC
- 26 58 and the QDC 58 is locked in a connected mode. When
- 27 the FPSO 10 arrives on site, the canister 16 is picked up
- and latched in, the bottom of the RHU 60 is latched into
- 29 it, the RHU 60 unbolted at the intermediate level, and
- 30 the whole stack from this unbolted level upwards is
- jacked up to give access for riser connection to riser
- 32 hangoff flanges. A pickup winch line 82 (or the line of
- a temporary small service crane) is deployed, taken down

33

1 through the canister 16 core, brought backup to the

- 2 surface, and connected to the first riser 20a, which will
- 3 have been raised to the surface and disconnected from the
- 4 temporary buoy. This activity requires the assistance of
- 5 another vessel; riser installation and replacement are
- 6 rare events. The assisting vessel then lowers the top of
- 7 the riser 20a until it is below the canister 16 and the
- 8 weight of the riser 20a is transferred to the pull-in
- 9 line 82. The riser 20a is then pulled up and the
- 10 hangoff flange is bolted up. This requires good access
- 11 for Hydratight (TM) bolting equipment and the operators,
- 12 hence the need to break the RHU 60 and jack it apart.
- 13 This process is repeated for each of the risers 20.

14

- 15 Turning now to Fig 17, there is shown a perspective view
- of a vessel 110 shown moored to an offshore mooring and
- 17 flowline system in accordance with an alternative
- 18 embodiment of the present invention, the system indicated
- generally by reference numeral 112. The system 112 is
- 20 essentially similar to the system 12 shown in Figs 1 to
- 11 16, and like components share the same reference numerals
- incremented by 100. The vessel 110 may be a similar
- vessel to that described above in relation to Figs 1 to
- 16, but will typically be an FSO. The system 112 differs
- from the system 12 in that it includes only a single
- riser 120 and associated jumper 132, and therefore does
- 27 not require the multiple path swivel stack 70 of the
- 28 system 12. Additionally, with only a single riser 20, an
- 29 indexing system may not be required.

- 31 Turning now to Fig 18, there is shown a side view of a
- 32 bow 226 of a vessel 210 shown moored to an offshore
- mooring and riser inboarding system in accordance with a

1 further alternative embodiment of the present invention,

- 2 the system indicated generally by reference numeral 212.
- The system 212 is essentially similar to the system 12
- 4 shown in Figs 1 to 16, and like components share the same
- 5 reference numerals incremented by 200. The vessel 210
- 6 shown in Fig 18 is a DP vessel such as an FPSO, and
- 7 includes thrusters (not shown) for maintaining the vessel
- 8 in a fixed geographical location. This enables the
- 9 vessel 210 to remain on station, that is, in the vicinity
- 10 of a buoy 216 forming a first mooring element of the
- 11 system 212. As the vessel 210 is dynamically positioned,
- it is not necessary for the buoy 216 to be moored
- 13 relative to the seabed 14 by heavy mooring lines such as
- the catenaries 34; this is because the buoy 216 does not
- 15 need to transmit loads experienced by the vessel 210 due
- 16 to the prevailing wind, wave or tide to the seabed 14.
- 17 Accordingly, risers 220 are able to maintain the buoy 216
- 18 approximately on station. However, the indexing system
- 19 may be utilised to account for friction in a swivel of
- the system 212; the indexing system may be activated to
- 21 maintain a rotational position (about the Y axis) of the
- 22 buoy 216. This ensures that the lower assembly does not
- turn with the weathervaning FPSO 210, which could result
- in the risers 220 twisting about each other and the
- individual risers 220 being subjected to excessive,
- detrimental twist. The risers 220 are thus maintained on
- 27 a constant geodetic azimuth. In this situation, the
- indexing motor will be controlled automatically by a
- 29 system of gyrocompasses and a computer (not shown), with
- 30 a manual override for emergency situations.

- 32 As shown in Fig 19, which is a view prior to connection
- of a second connector 218 to the buoy 216, the inherent

35

1 buoyancy of the buoy 216 is such that the buoy is

- 2 initially below the sea surface 19, and a marker buoy 284
- 3 indicates the location of the primary buoy 216. By
- 4 locating the buoy 216 below the sea surface 19, the buoy
- 5 is shielded from external loads at surface. The system
- 6 212 is otherwise of similar construction and operation to
- 7 the system 12 of Figs 1 to 16.

8

- 9 Turning now to Fig 20, there is shown a side view of a
- 10 bow 326 of a vessel 310 shown moored to an offshore
- 11 mooring and riser inboarding system in accordance with a
- 12 still further alternative embodiment of the present
- invention, the system indicated generally by reference
- 14 numeral 312. The system 312 is essentially similar to
- the system 12 shown in Figs 1 to 16, and like components
- 16 share the same reference numerals incremented by 300.
- 17 However, in a similar fashion to the system 212 of Figs
- 18 and 19, the vessel 310 is a DP vessel. Accordingly,
- 19 the first mooring element of the system 312, which takes
- the form of a canister 316 (similar to the canister 16 of
- the system 12) does not need to be moored relative to the
- seabed 14 by heavy mooring lines; the risers 320 are able
- 23 to maintain the canister 316 approximately on station.

- 25 As shown in Fig 21, which is a view prior to connection
- of a second connector 318 to the canister 316, the
- inherent buoyancy of the canister 316 is such that the
- 28 canister is initially at a similar level to the canister
- 29 16. However, the canister 316 may be initially below sea
- 30 surface 19, in a similar fashion to the buoy 216 of the
- 31 system 212, if desired.

Various modifications may be made to the foregoing
without departing from the spirit and the scope of the
present invention.

4

5 For example, the above described embodiments of the

- 6 invention include adjustable couplings in the form of
- 7 inner and outer gimbal members which facilitate relative
- 8 rotation between the vessel and the first mooring element
- 9 about two axes of rotation. However, the system may
- 10 include any suitable, alternative adjustable couplings in
- 11 place of the gimbals.

12

- 13 The system may comprise any suitable riser found in the
- 14 offshore environment, used in the oil and gas exploration
- 15 and production industry, for bringing the riser onboard
- 16 or inboard to a vessel.

17

- 18 In the embodiments of the invention where a DP vessel is
- moored using the system, the vessel may weathervane
- 20 around the first mooring element, rotating about a
- 21 vertical or Y axis, with little or minimal rotation about
- the other axes of rotation. By allowing the vessel to
- 23 weathervane, loads on the vessel may be reduced.

- 25 The first and second mooring elements may be coupled
- 26 together using any, suitable alternative coupling/locking
- 27 mechanism.

Claims

- 1. An offshore vessel mooring and riser inboarding system, the system comprising:
- a first mooring element for locating in an offshore environment and coupled to a fluid flow riser, the first mooring element defining an internal passage for receiving the riser;
- a connector assembly comprising:
 - a cantilever support for mounting on a vessel;
 - a second mooring element suspended from the cantilever support and which can be connected to the first mooring element, to thereby suspend the first mooring element from the cantilever support,

an outer gimbal member mounted for rotation relative to the cantilever support,

an inner gimbal member mounted within and for rotation relative to the outer gimbal member, and

a rotatable coupling for facilitating rotation of the inner gimbal member relative to the first mooring element;

a transfer flowline adapted to be coupled to the fluid flow riser; and

a swivel, the transfer flowline coupled to the fluid flow riser through the swivel;

wherein in use:

the first and second mooring elements are connected to facilitate coupling of the fluid flow riser and the transfer flowline;

the outer gimbal member, the inner gimbal member and the rotatable coupling together permit relative rotation between the vessel and the first

mooring element about three mutually perpendicular axes of rotation;

the fluid flow riser ascends through the internal passage in the first mooring element, through the rotatable coupling and the inner and outer gimbals to the swivel; and

the rotatable coupling and the swivel facilitate unlimited rotation of the vessel about a longitudinal axis of the first mooring element while maintaining fluid flow.

- 2. A system as claimed in claim 1, wherein the fluid flow riser and the transfer flowline, when coupled together, extend through an aperture in the inner gimbal member.
- 3. A system as claimed in claim 1 or 2, wherein the three mutually perpendicular axes of rotation are taken with reference to the first mooring element in a neutral position.
- 4. A system as claimed in any one of claims 1 to 3, wherein the riser is a conduit for hydrocarbon containing fluids.
- 5. A system as claimed in claim 1, 2 or 3, further comprising at least one further riser, wherein the further riser comprises one or more cables selected from the group consisting of a power cable and a control cable.
- 6. A system as claimed in claim 1, 2 or 3, further comprising at least one further riser, wherein the

further riser comprises one or more cables selected from the group consisting of an electrical cable and a hydraulic cable.

- 7. A system as claimed in any one of claims 1 to 4, further comprising at least one further riser, wherein the further riser is an umbilical.
- 8. A system as claimed in claim 1, wherein connection of the first and second mooring elements facilitates flow of fluid between the fluid flow riser, the transfer flowline and the vessel.
- 9. A system as claimed in claim 8, wherein the transfer flowline is for the passage of fluid from the fluid flow riser into the transfer flowline and to the vessel, or vice-versa.
- 10. A system as claimed in claim 5 or 6, further comprising a transfer line, which provides one or more connections selected from the group consisting of an electrical connection and a hydraulic connection to the riser.
- 11. A system as claimed in claim 10, wherein the transfer line facilitates one or more of power supply, data transmission, and supply of hydraulic control fluid.
- 12. A system as claimed in any one of claims 1 to 11, wherein the second mooring element is adapted to be mounted for movement relative to the cantilever support.

Patent

- 13. A system as claimed in claim 12, wherein the cantilever support is located extending beyond a bow of the vessel.
- 14. A system as claimed in any one of claims 1 to 13, wherein the inner gimbal member is rotatable about an inner gimbal axis and the outer gimbal member about an outer gimbal axis.
- 15. A system as claimed in claim 14, wherein the inner and outer gimbal member axes are disposed substantially perpendicular to one another.
- 16. A system as claimed in any one of claims 1 to 15, wherein the rotatable coupling facilitates rotation between the inner gimbal member and the second mooring element, to thereby permit relative rotation between the vessel and the first mooring element about one of the three axes of rotation.
- 17. A system as claimed in claim 16, wherein the rotatable coupling is provided between the inner gimbal member and the second mooring element.
- 18. A system as claimed in any one of claims 1 to 17, wherein the rotatable coupling facilitates rotation between the second mooring element and the first mooring element, thereby permitting relative rotation between the vessel and the first mooring element about one of the three axes of rotation.

- 19. A system as claimed in claim 18, wherein the rotatable coupling is provided between the first and second mooring elements.
- 20. A system as claimed in any one of claims 1 to 19, wherein the rotatable coupling is a swivel.
- 21. A system as claimed in any one of claims 1 to 20, wherein the inner and outer gimbal members are annular rings.
- 22. A system as claimed in any one of claims 1 to 21, wherein the inner gimbal member is mounted to the outer gimbal member by inner trunnions, and wherein the outer gimbal member is mounted to the cantilever support by outer trunnions, the trunnions of the inner gimbal member disposed perpendicular to those of the outer gimbal member.
- 23. A system as claimed in any one of claims 1 to 22, wherein the connector assembly is releasably mountable on the vessel.
- 24. A system as claimed in any one of claims 1 to 23, wherein the first mooring element is buoyant.
- 25. A system as claimed in any one of claims 1 to 24, wherein the system comprises a buoyant member, and wherein the first mooring element is indirectly coupled to the buoyant member.
- 26. A system as claimed in any one of claims 1 to 25, wherein the first mooring element is tubular.

Doc. No.: 157-1 CA/PCT

Patent

- 27. A system as claimed in any one of claims 1 to 26, wherein the first mooring element is adapted to be located at surface prior to connection of the first and second mooring elements together.
- 28. A system as claimed in any one of claims 1 to 26, wherein the entire first mooring element is adapted to be located below sea surface level prior to connection of the first and second mooring elements together.
- 29. A system as claimed in claim 28, wherein a location of the first mooring element prior to connection is indicated by a marker buoy.
- 30. A system as claimed in claim 25, wherein the buoyant member is adapted to be located at sea surface prior to connection of the first and second mooring elements together.
- 31. A system as claimed in any one of claims 1 to 30, wherein the first mooring element is adapted to be moored to a seabed in the offshore environment by a plurality of mooring lines, the mooring lines adapted to bear loading of the vessel on the first mooring element, to maintain the element on station and to minimise transmission of loads to the main flowline.
- 32. A system as claimed in claim 31, wherein the mooring lines are coupled to a lower portion of the first mooring element.

- 33. A system as claimed in any one of claims 1 to 30, wherein the system is for a dynamically positionable vessel.
- 34. A system as claimed in any one of claims 1 to 33, wherein the first and second mooring elements define respective first and second connector elements.
- 35. A system as claimed in claim 34, wherein the first and second mooring elements are adapted to be coupled together in a quick-connect and disconnect arrangement.
- 36. A system as claimed in any one of claims 1 to 35, wherein one of the first and second mooring elements comprises a male member and the other a female member, the female member adapted to receive the male member for engagement of the elements.
- 37. A system as claimed in any one of claims 1 to 36, wherein the connector assembly comprises a locking arrangement for locking the first and second mooring elements together.
- 38. A system as claimed in claim 37, wherein the locking arrangement comprises at least one latch adapted to provide a releasable locking engagement between the first and second mooring elements.
- 39. A system as claimed in any one of claims 1 to 38, wherein the connector assembly comprises an intermediate connector for coupling the first and second mooring elements together.

- 40. A system as claimed in claim 39, wherein the intermediate connector is secured to the first mooring element and thus provided as part of the first mooring element, and is adapted to be releasably coupled to the second mooring element.
- 41. A system as claimed in claim 40, wherein the intermediate connector is also adapted to be releasably connected to the first mooring element.
- 42. A system as claimed in any one of claims 39 to 41, wherein the intermediate connector is adapted to support the riser, and defines a riser hang-off unit.
- 43. A system as claimed in claim 42, wherein the connector assembly comprises a jacking device for raising part of the connector assembly to provide access space for connection of the riser to the riser hang-off unit.
- 44. A system as claimed in any one of claims 1 to 43, comprising a plurality of fluid flow risers and a corresponding plurality of transfer flowlines, each transfer flowline associated with a corresponding fluid flow riser.
- 45. A system as claimed in claim 44, wherein each riser is associated with a separate well, for the flow of well fluids to the vessel.
- 46. A system as claimed in any one of claims 1 to 45, wherein the swivel is coupled to the second mooring element.

- 47. A system as claimed in any one of claims 1 to 46, wherein rotation between the vessel and the first mooring element about at least one of the other two of said axes of up to 60° from a neutral position is permitted, providing up to 120° total permissible rotation.
- 48. A system as claimed in any one of claims 1 to 47, comprising a device for adjusting an orientation of the second mooring element relative to the first mooring element, to facilitate connection of the elements.
- 49. A system as claimed in claim 48, wherein the connector assembly comprises an indexing device for adjusting a rotational orientation of the first and second mooring elements.
- 50. A system as claimed in claim 49, wherein the device is for adjusting at least one of a rotational position of the outer gimbal member relative to the support; and of the inner gimbal member relative to the outer gimbal member.
- 51. A method of mooring a vessel in an offshore environment, the method comprising the steps of:
 - locating a first mooring element in an offshore environment;
 - coupling a fluid flow riser to the first mooring element;
 - suspending a second mooring element of a connector assembly from a cantilever support mounted on a vessel;

raising the first mooring element and connecting it to the second mooring element to thereby suspend the first mooring element from the cantilever support, the second mooring element being connected to the first mooring element such that relative rotation between the vessel and the first mooring element about three mutually perpendicular axes of rotation is permitted, said relative rotation being permitted by means of an outer gimbal member mounted for rotation relative to the cantilever support, an inner gimbal member mounted within and for rotation relative to the outer gimbal member, and a rotatable coupling which facilitates rotation of the inner gimbal member relative to the first mooring element;

locating the fluid flow riser such that it ascends
through an internal passage in the first
mooring element, through the rotatable coupling
and the inner and outer gimbals to a swivel
coupled to the second mooring element; and
connecting a transfer flowline to the fluid flow
riser through the swivel;

wherein the rotatable coupling and the swivel facilitate unlimited rotation of the vessel about a longitudinal axis of the first mooring element while maintaining fluid flow.

52. A method as claimed in claim 51, comprising transferring fluid between the fluid flow riser, the transfer flowline and the vessel.

53. A connector assembly for an offshore vessel mooring and riser inboarding system of a type comprising: a first mooring element located in an offshore environment and coupled to a fluid flow riser, the first mooring element defining an internal passage for receiving the riser; and a transfer flowline adapted to be coupled to the fluid flow riser;

the connector assembly comprising:

- a cantilever support for mounting on a vessel;
- a second mooring element suspended from the cantilever support for connecting to the first mooring element, to thereby facilitate coupling of the riser and the transfer line and suspension of the first mooring element from the cantilever support;
- an outer gimbal member mounted for rotation relative to the cantilever support;
- an inner gimbal member mounted within and for rotation relative to the outer gimbal member;
- a rotatable coupling for facilitating rotation of the inner gimbal member relative to the first mooring element; and
- a swivel through which the transfer flowline is couplable to the fluid flow riser;

wherein, in use:

- the first and second mooring elements are connected to facilitate coupling of the fluid flow riser and the transfer flowline;
- the outer gimbal member, the inner gimbal member and the rotatable coupling together

Patent

permit relative rotation between the vessel and the first mooring element about three mutually perpendicular axes of rotation;

- the fluid flow riser ascends through the internal passage in the first mooring element, through the rotatable coupling and the inner and outer gimbals to the swivel; and
- the rotatable coupling and the swivel facilitate unlimited rotation of the vessel about a longitudinal axis of the first mooring element while maintaining fluid flow.

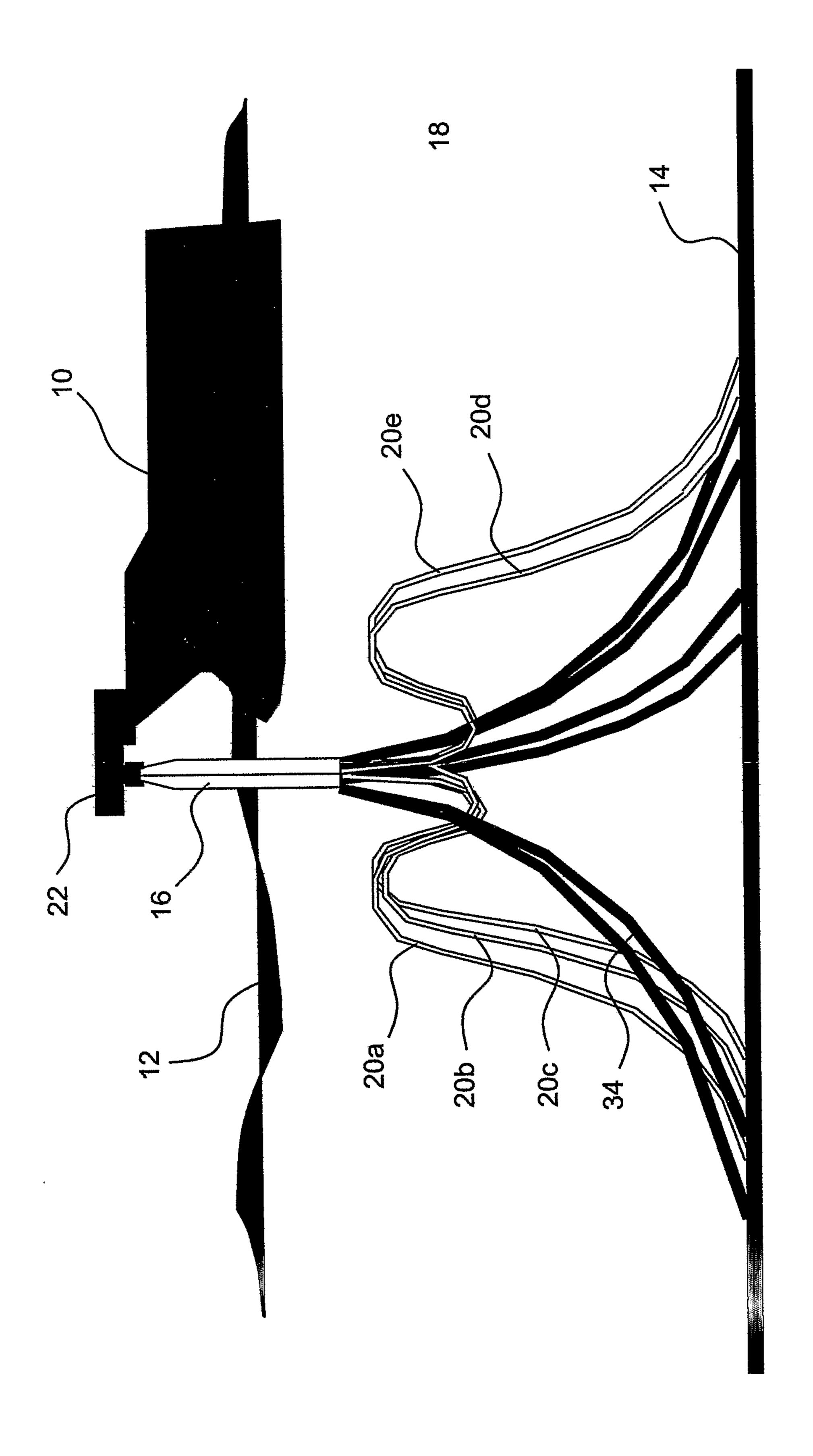


Figure 1.

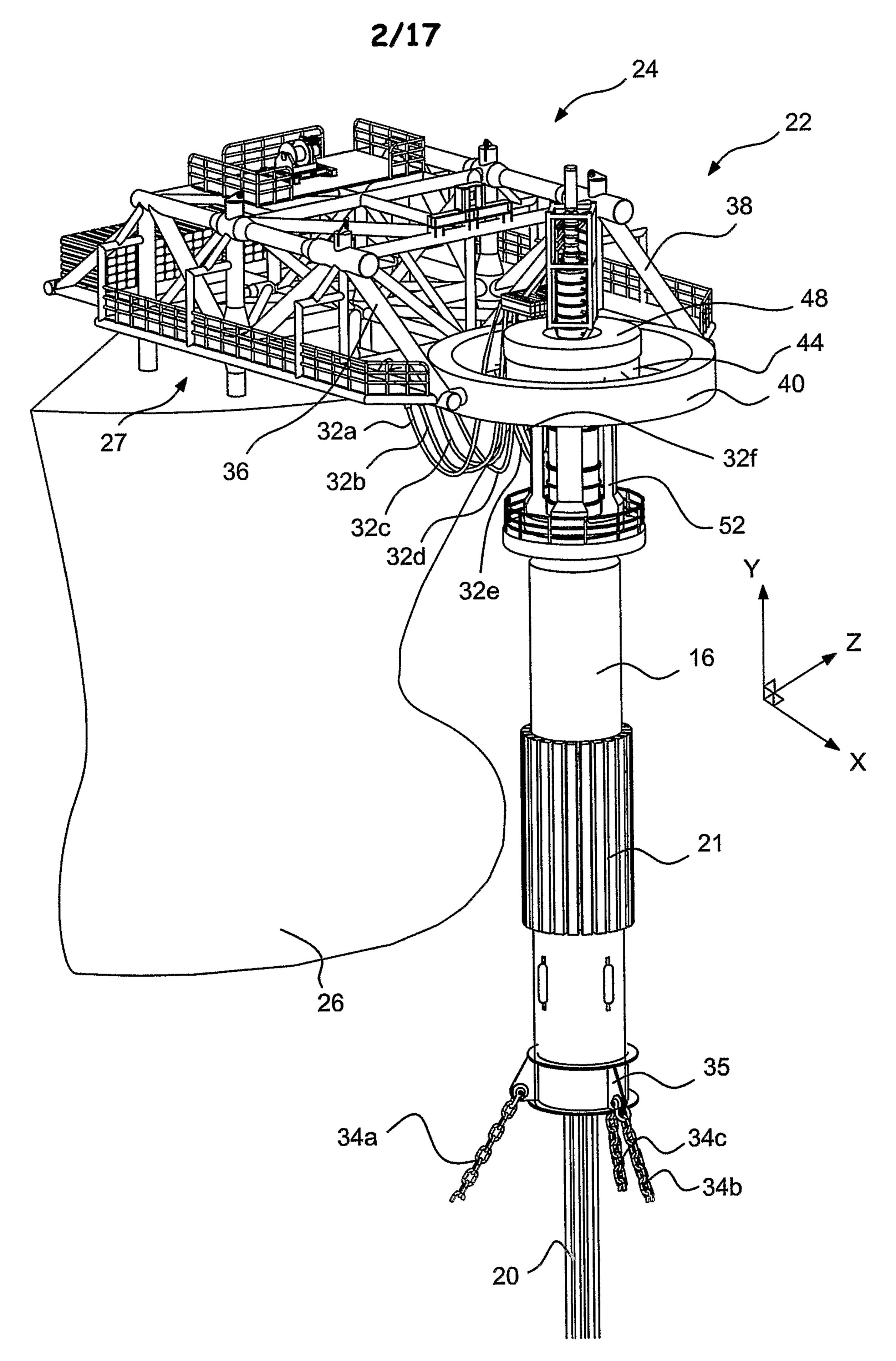


Figure 2.

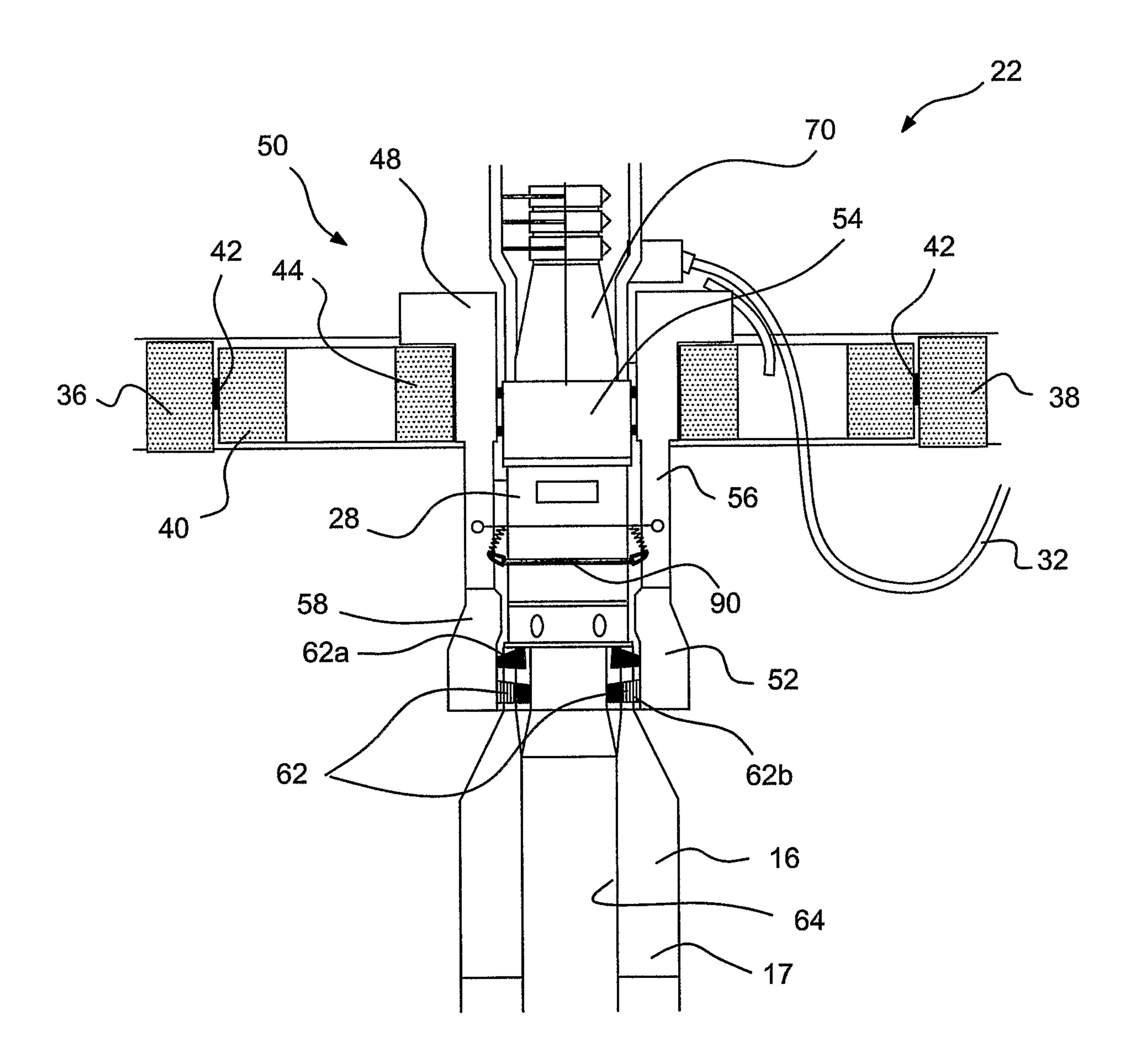
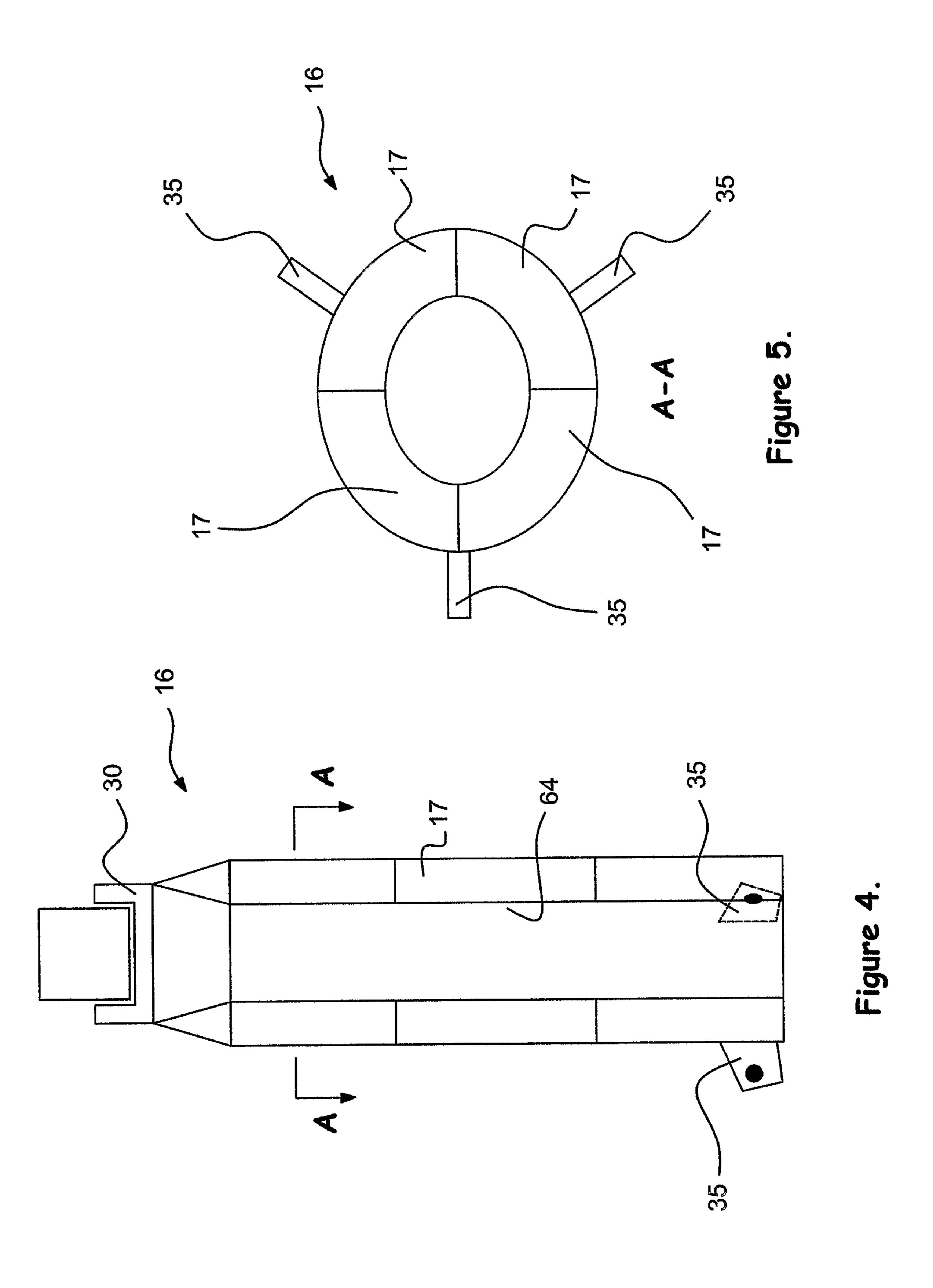
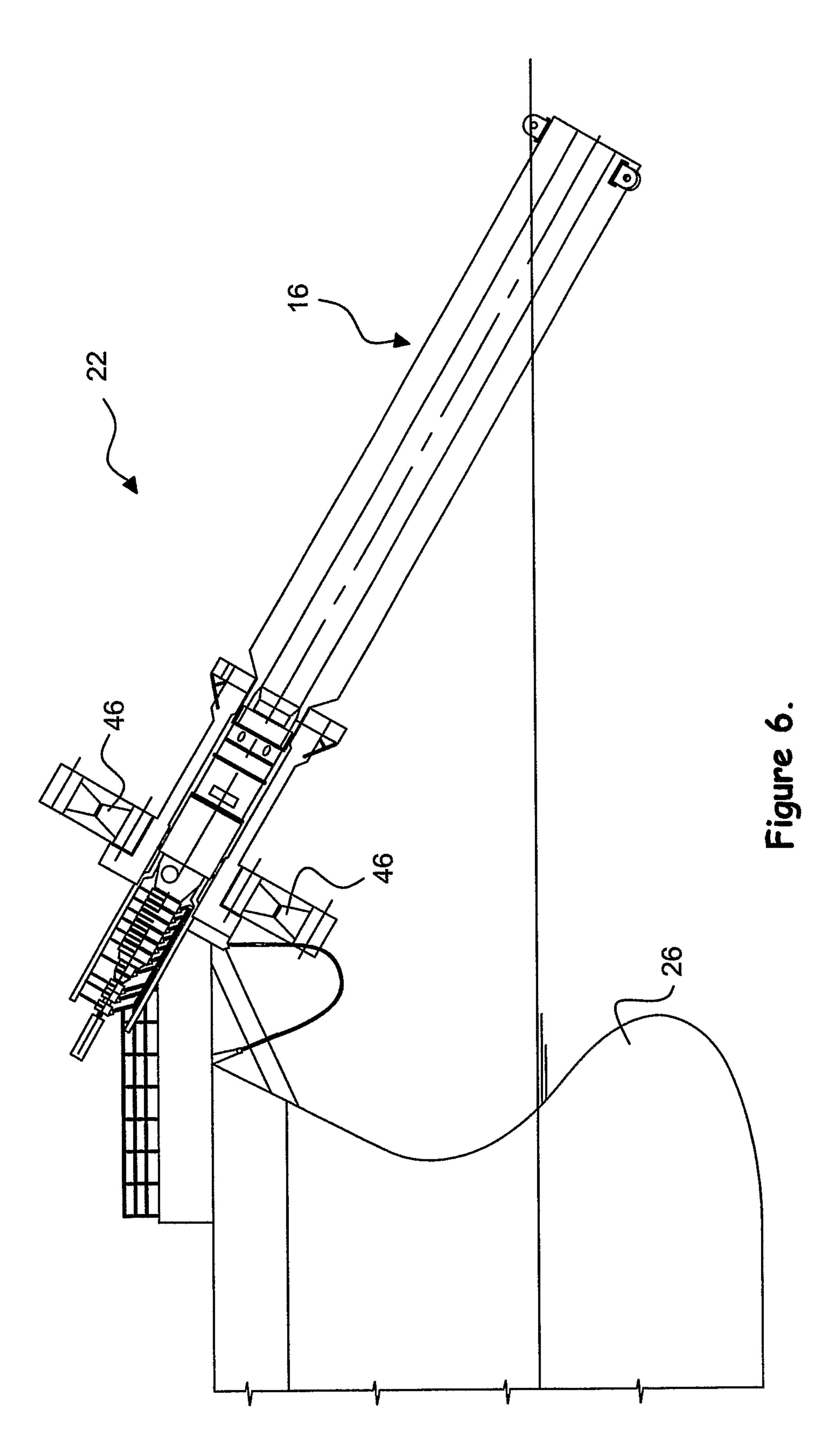


Figure 3.

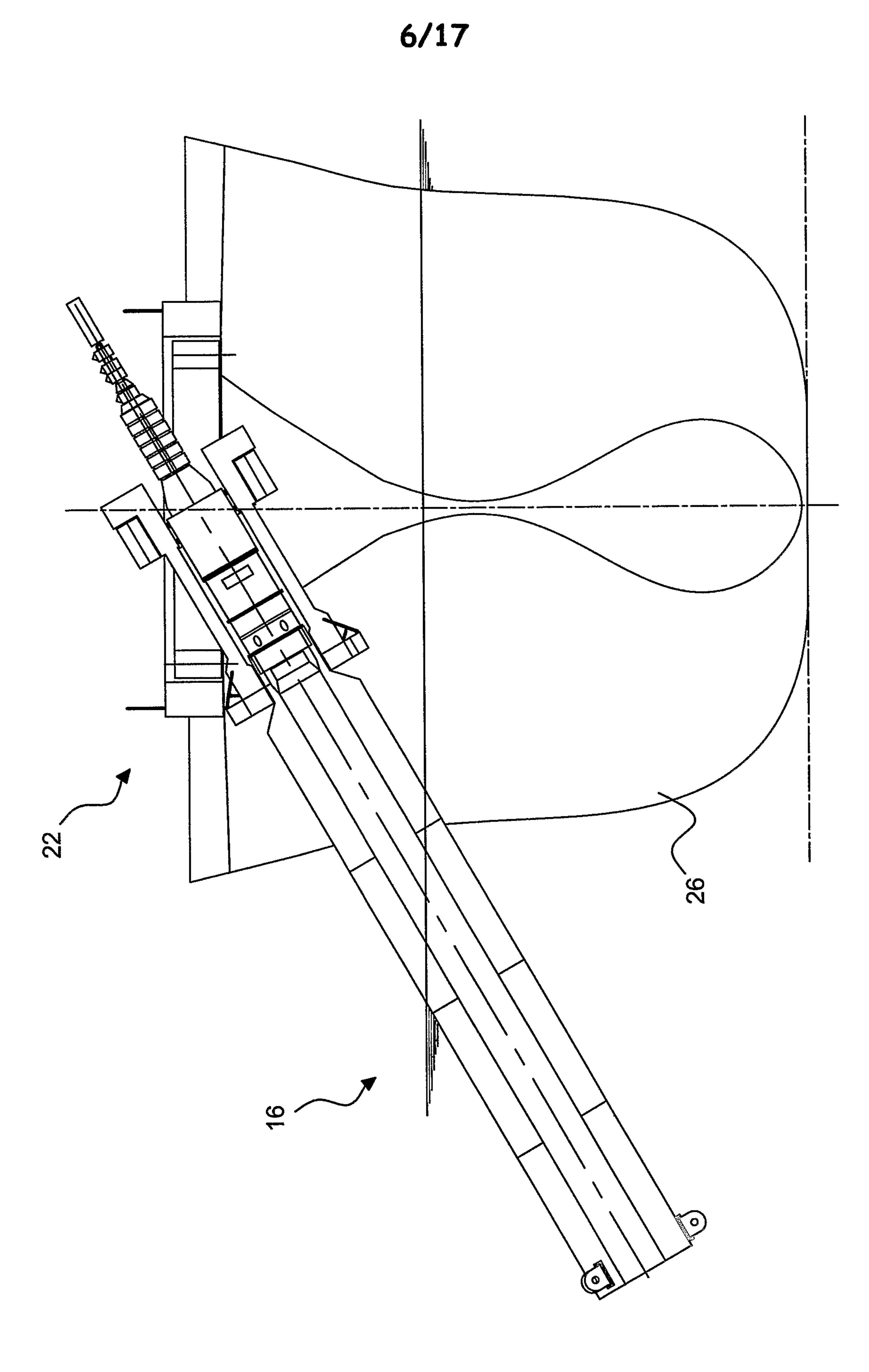
4/17











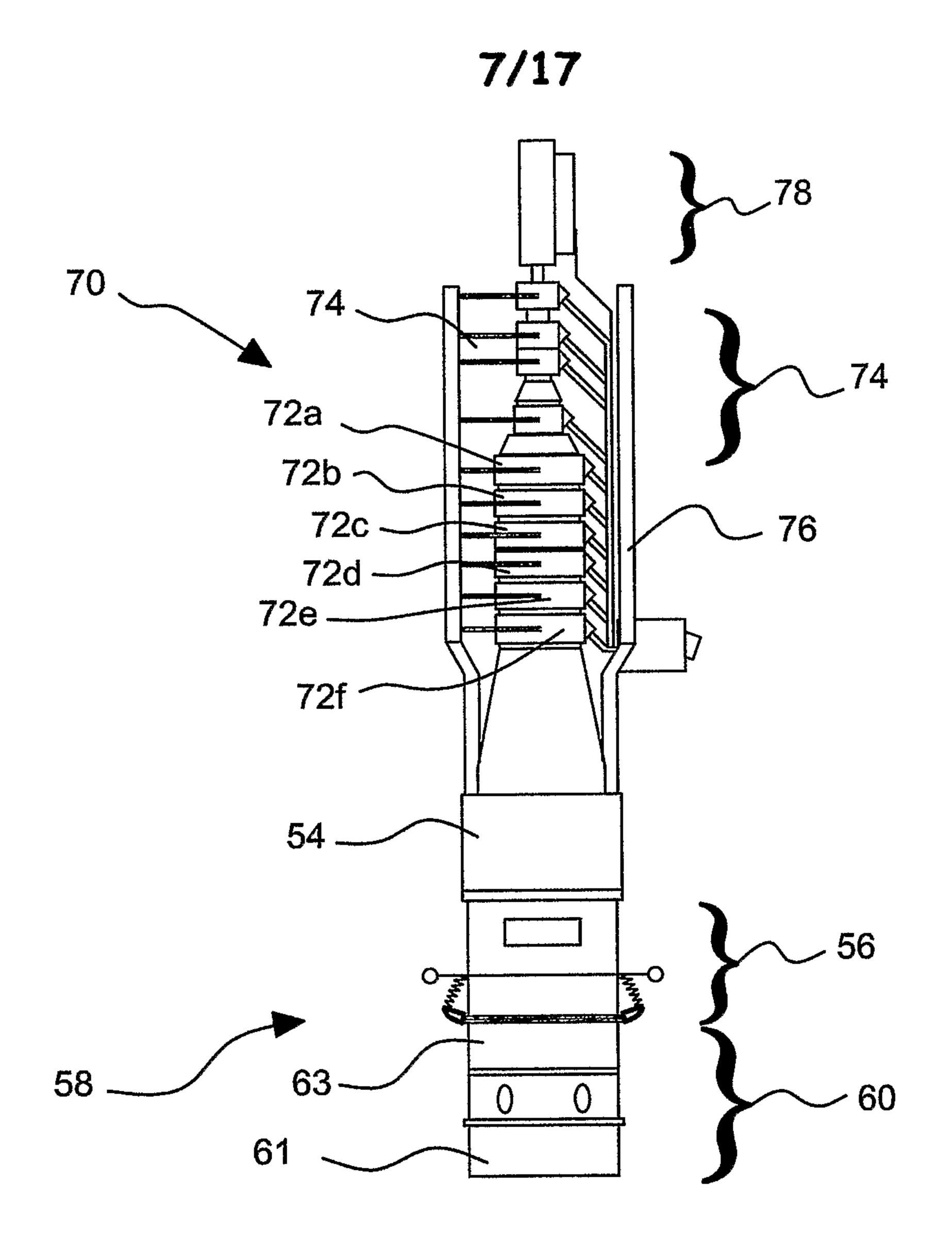


Figure 8.

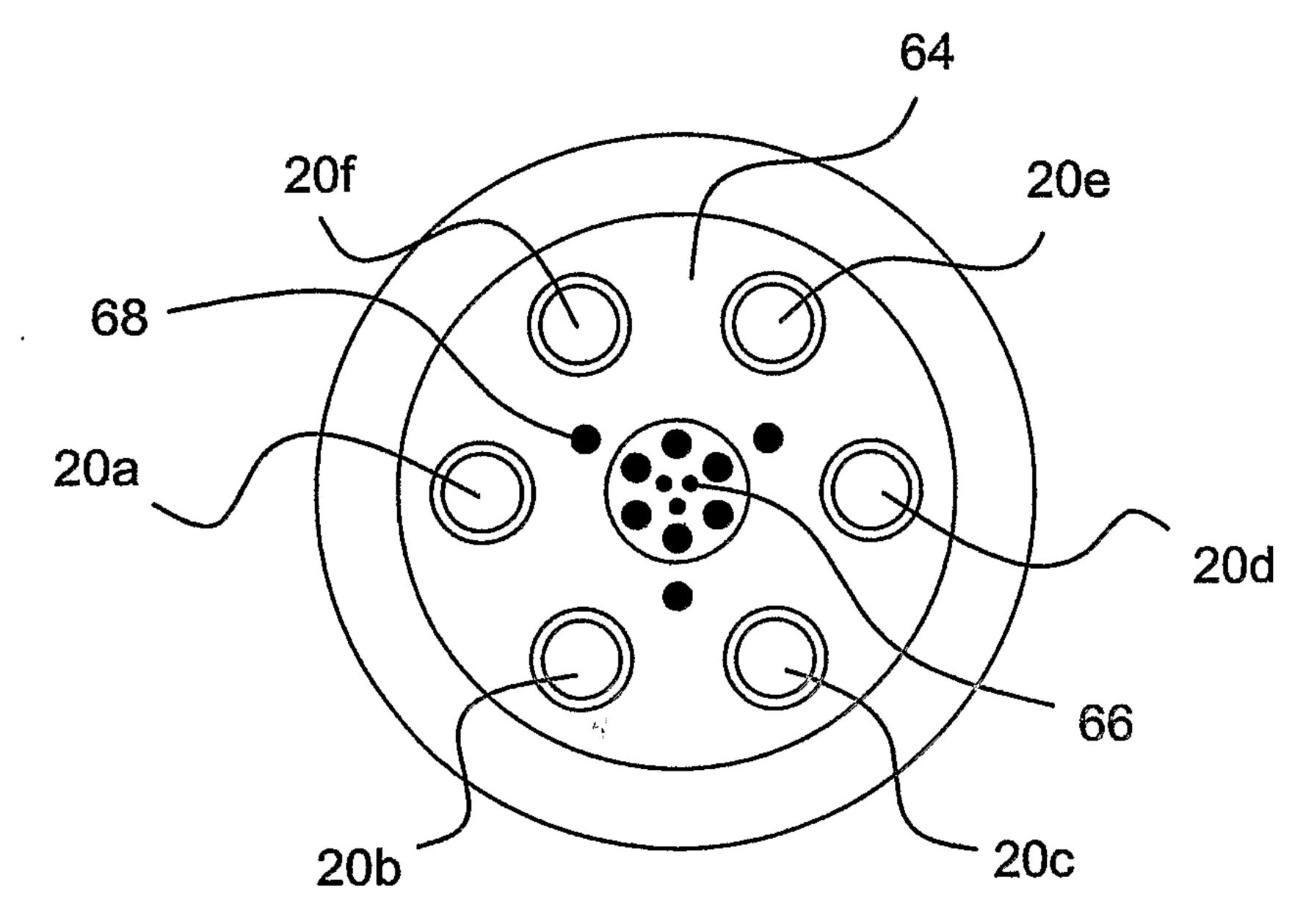


Figure 9.

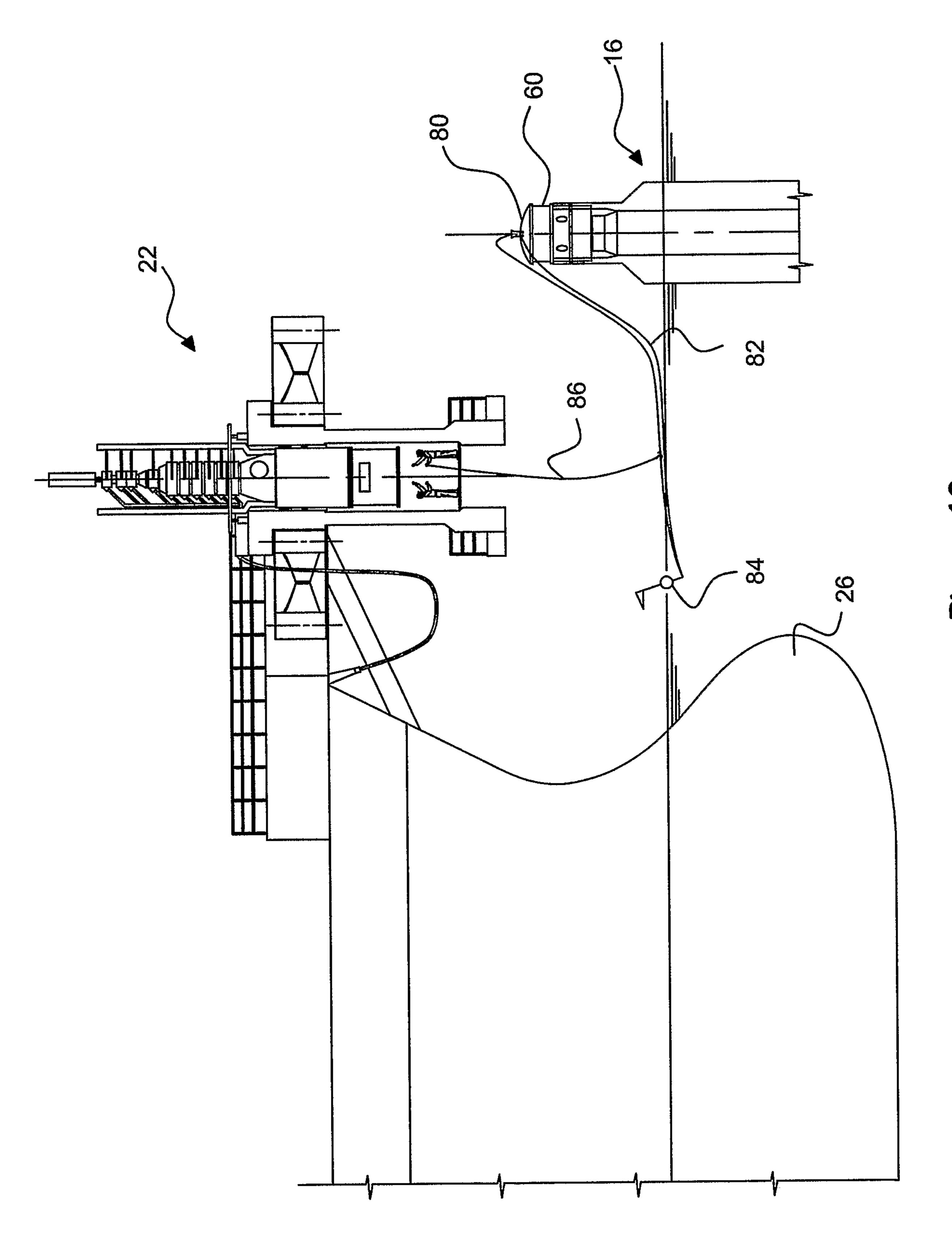
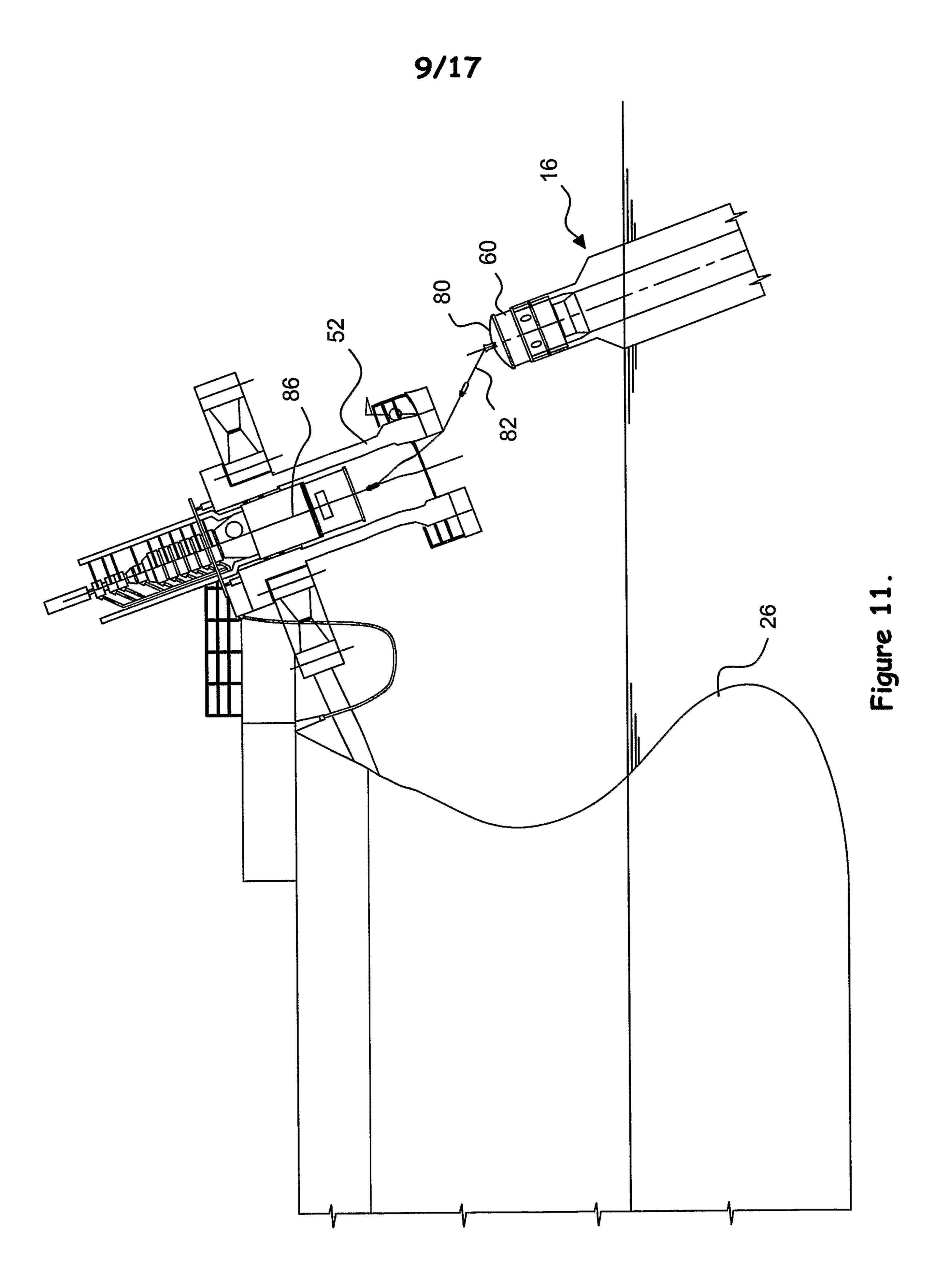
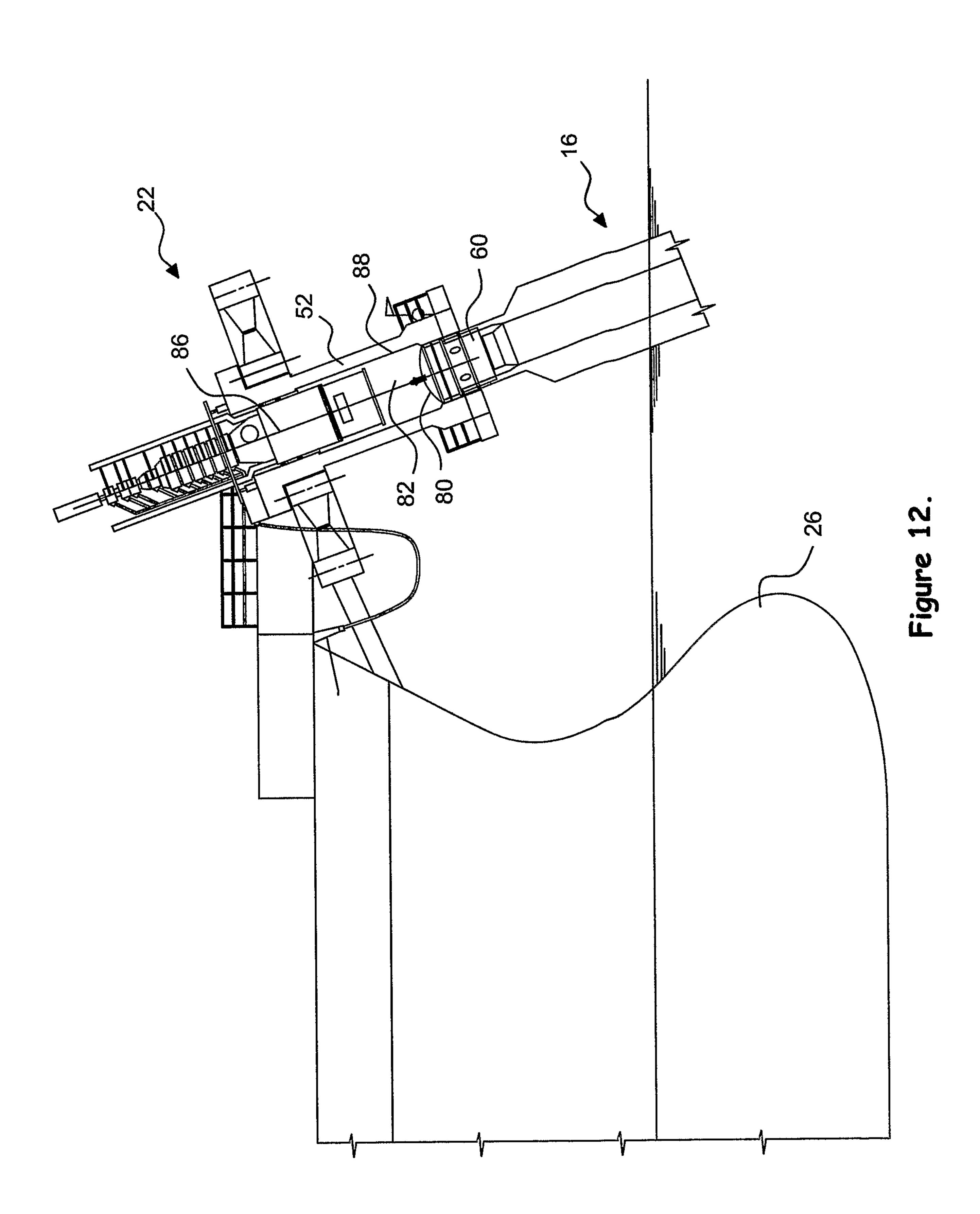


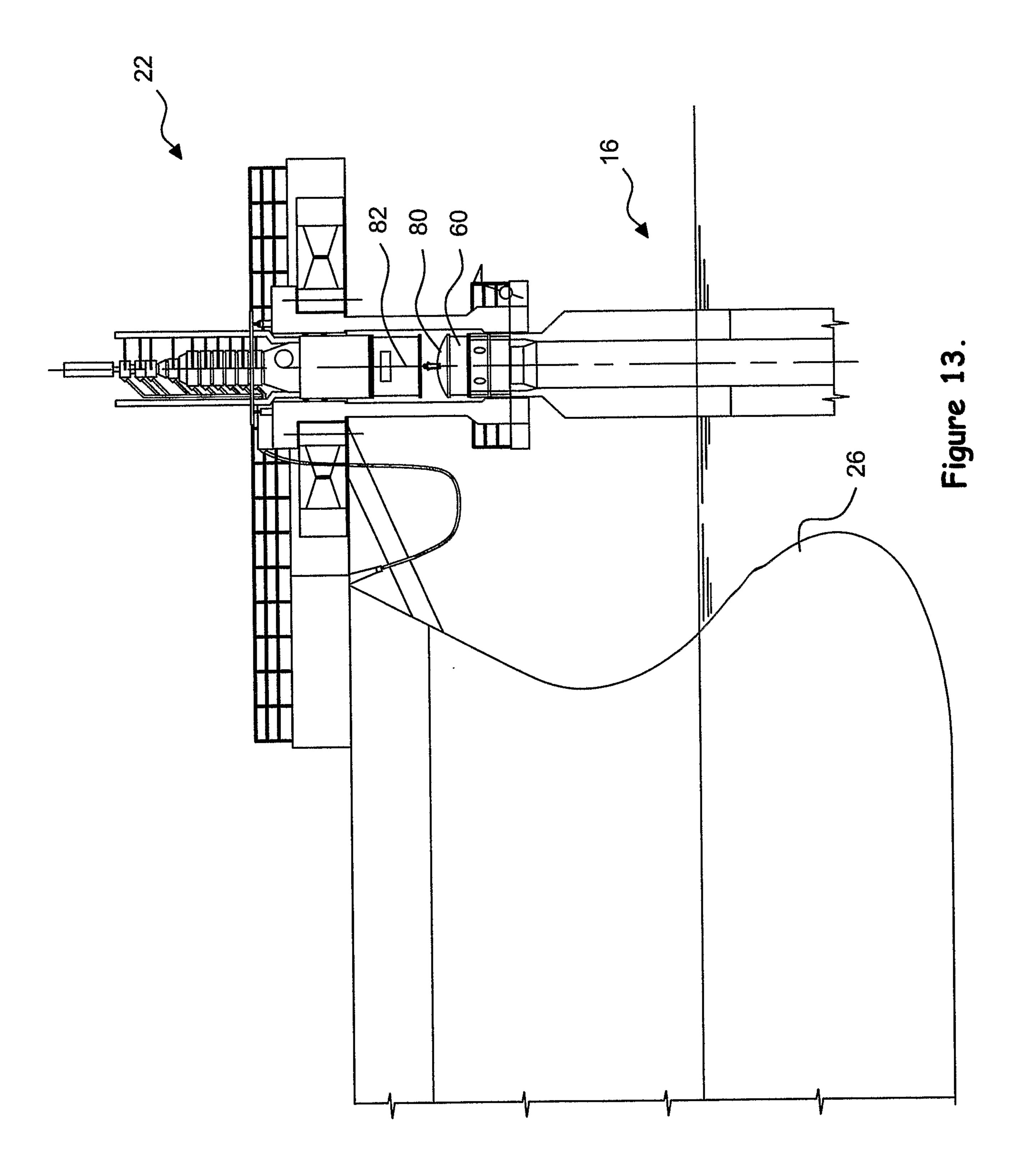
Figure 10.



10/17



11/17



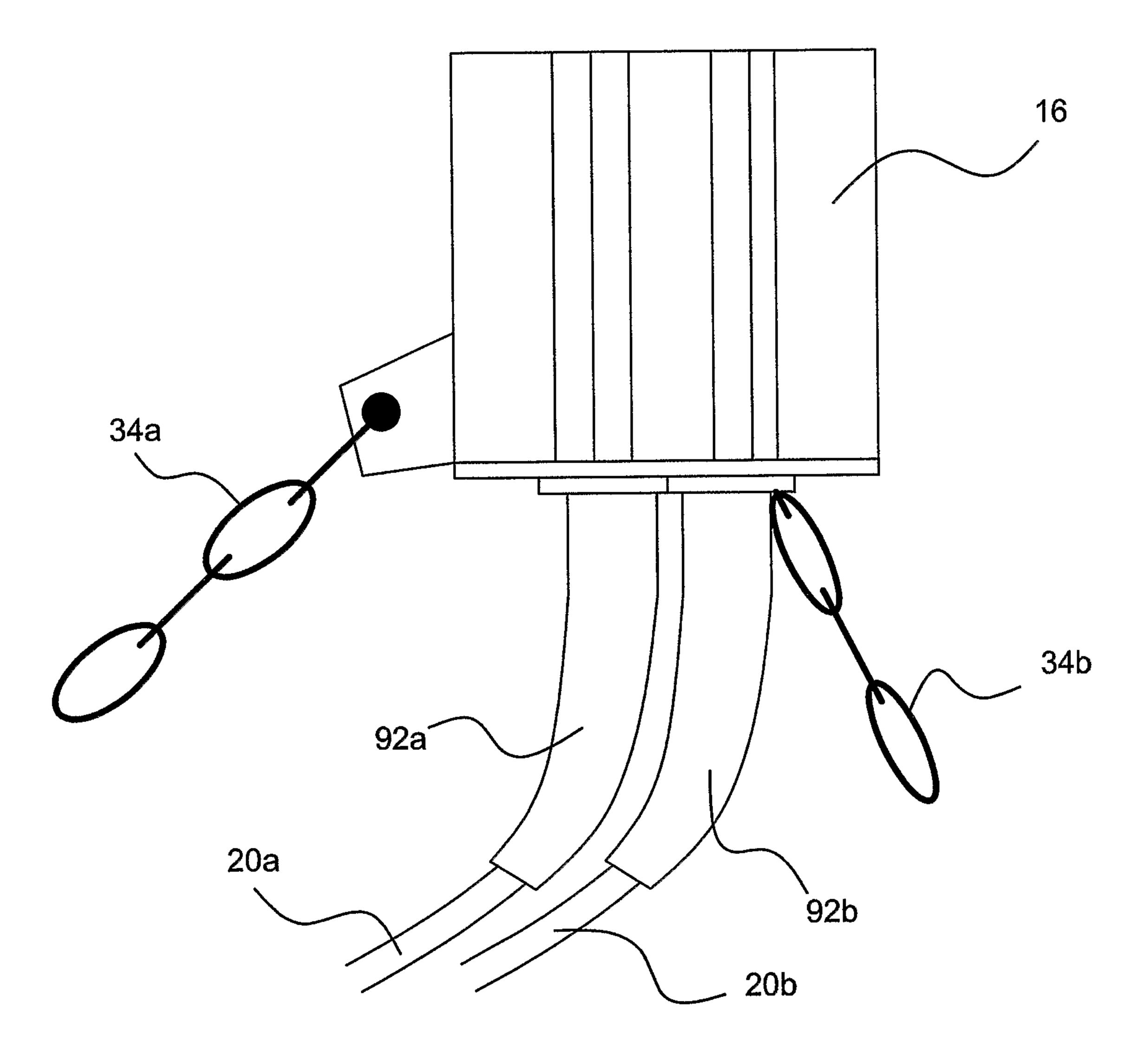


Figure 14.

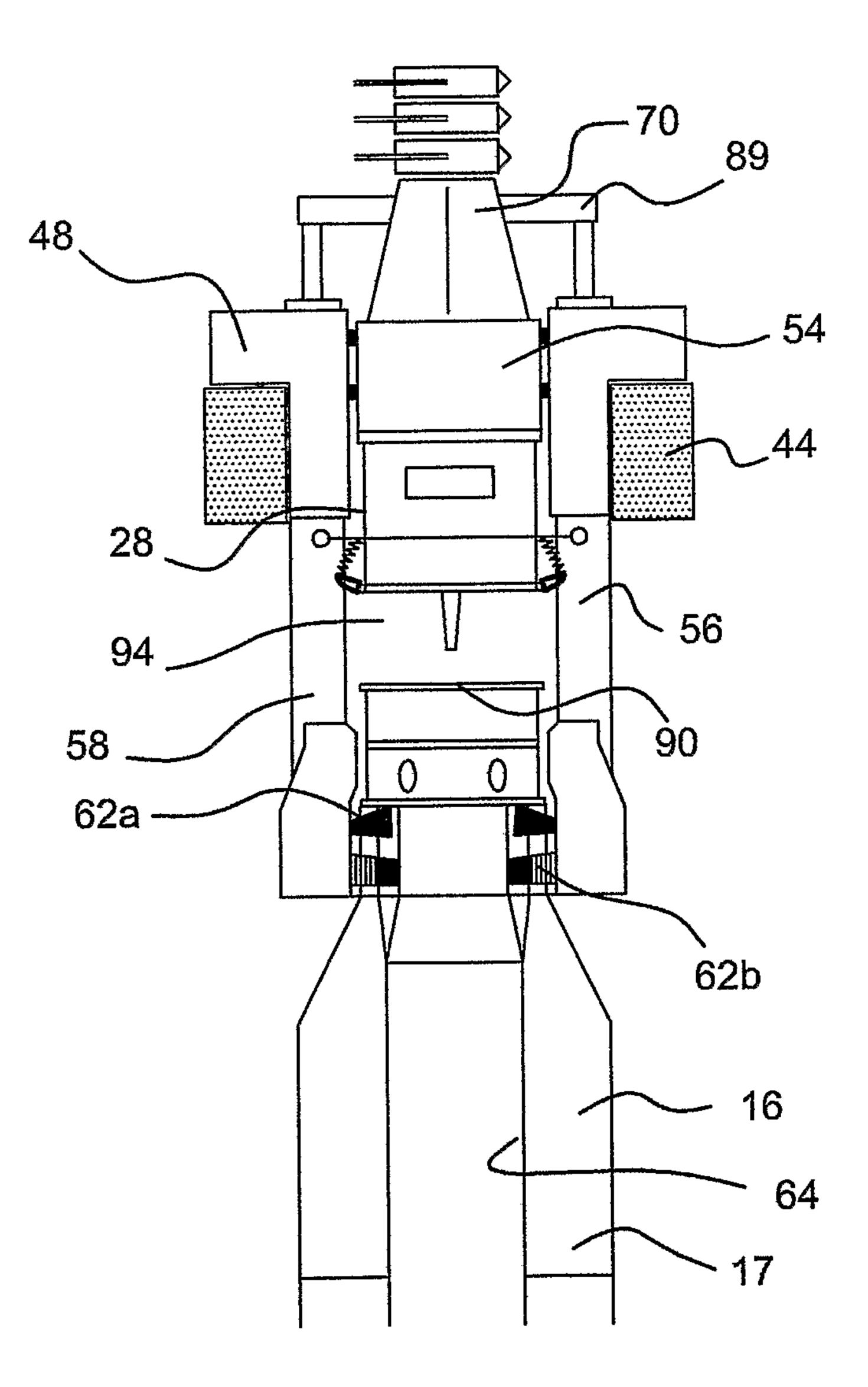


Figure 15.

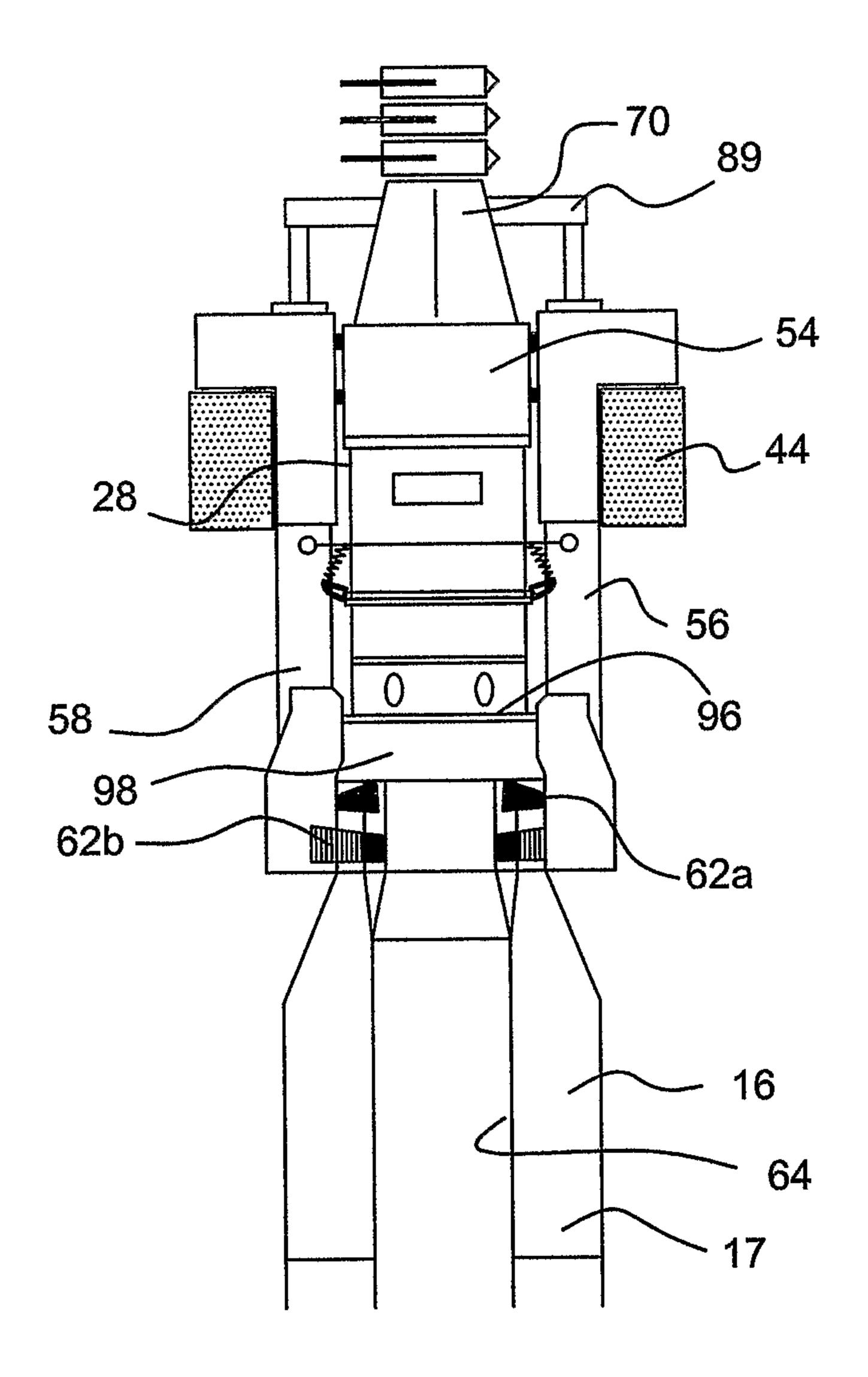
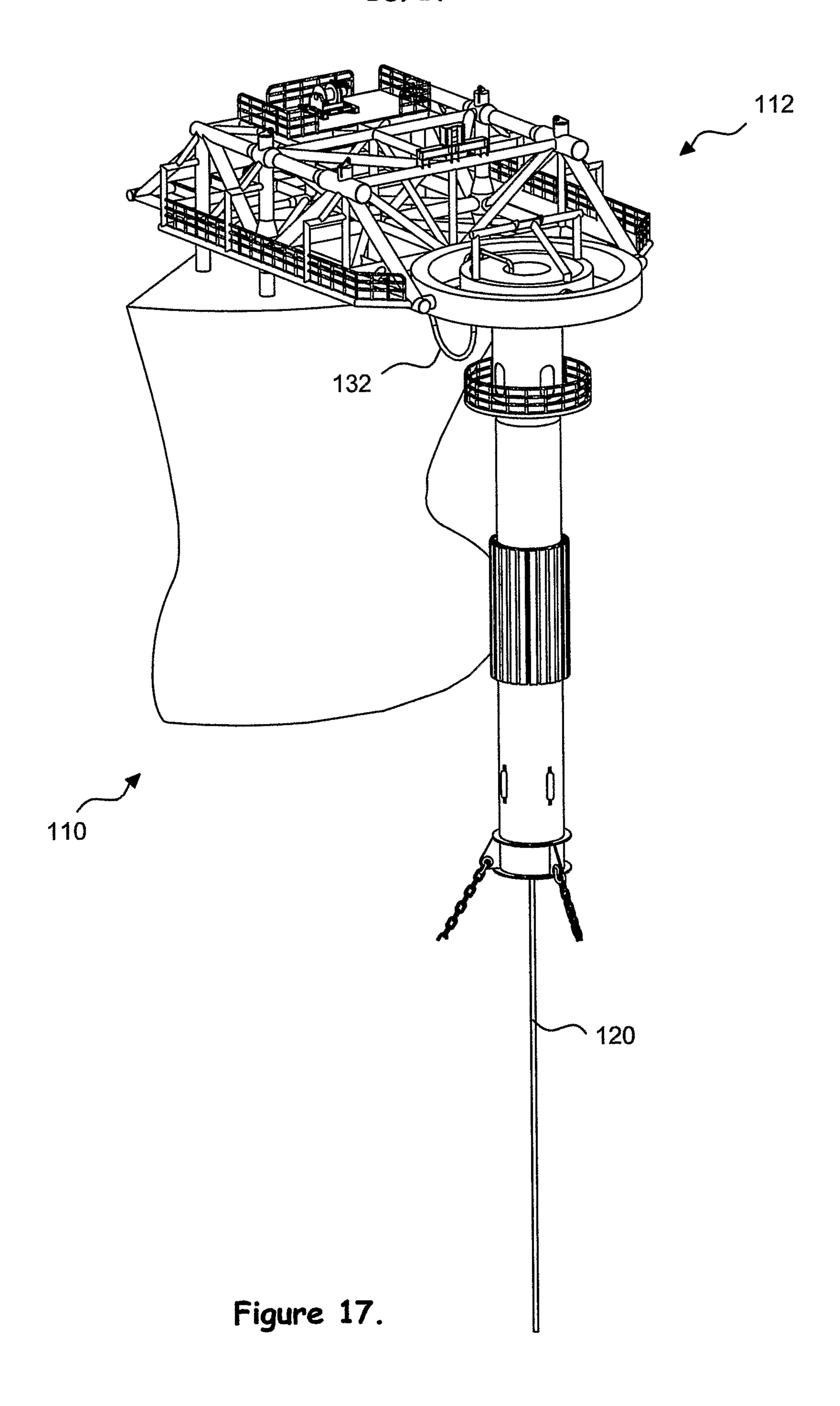


Figure 16.



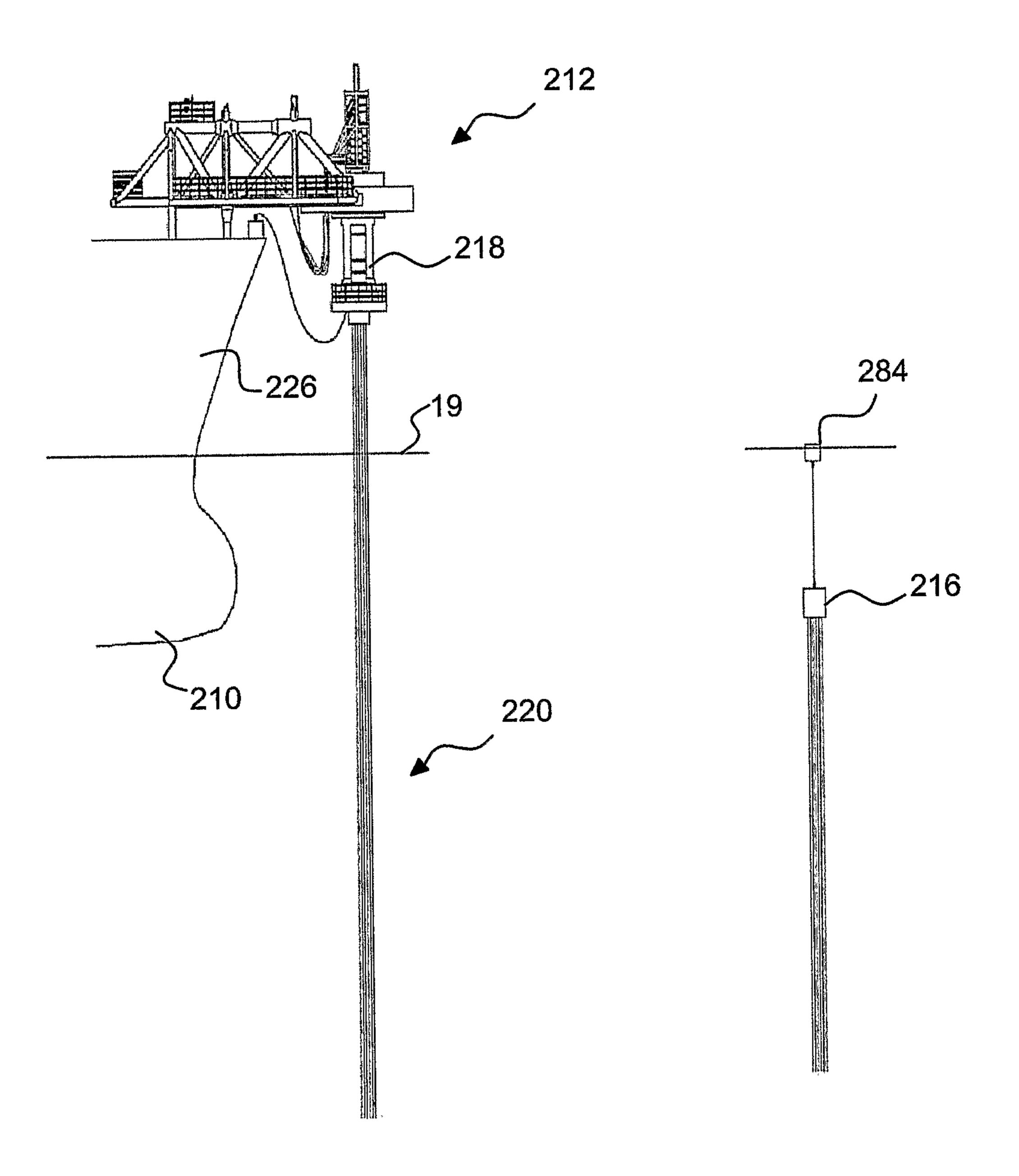


Figure 18.

Figure 19.

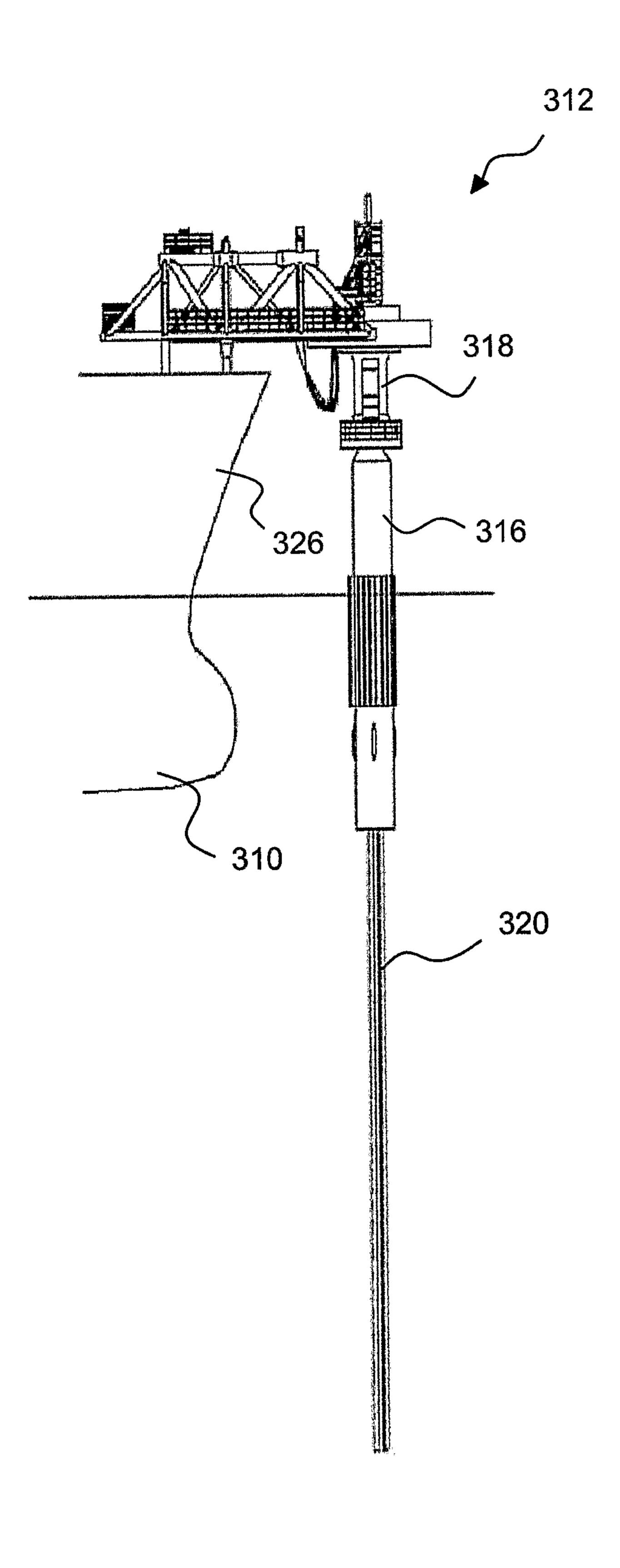


Figure 20.

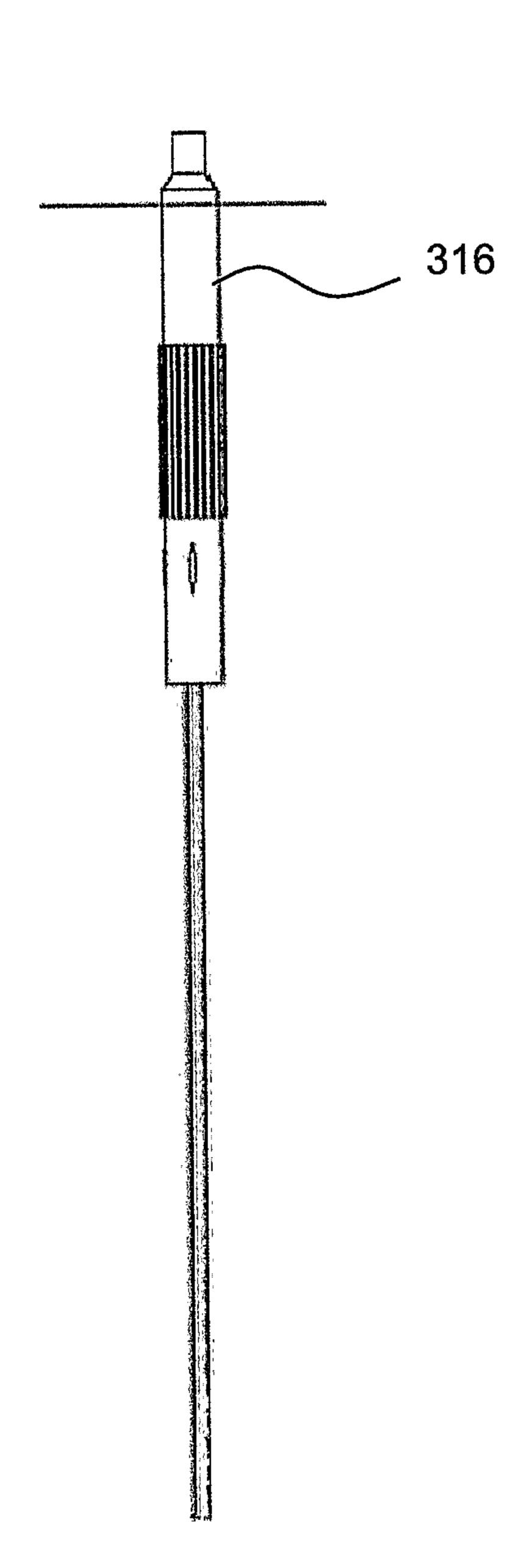


Figure 21.

