

(12) **United States Patent**  
**Strobel**

(10) **Patent No.:** **US 10,203,068 B2**  
(45) **Date of Patent:** **Feb. 12, 2019**

(54) **METHOD AND DEVICE FOR PRECOOLING A CRYOSTAT**

USPC ..... 62/51.1  
See application file for complete search history.

(71) Applicant: **Bruker BioSpin GmbH**, Rheinstetten (DE)

(56) **References Cited**

(72) Inventor: **Marco Strobel**, Karlsruhe (DE)

U.S. PATENT DOCUMENTS

(\* ) Notice: Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 0 days.

4,771,823 A \* 9/1988 Chan ..... F17C 13/00  
165/61  
5,111,665 A \* 5/1992 Ackermann ..... F25D 19/006  
505/894  
5,934,082 A 8/1999 Steinmeyer  
6,305,174 B1 \* 10/2001 Binneberg ..... B64G 1/50  
62/383

(21) Appl. No.: **15/225,094**

(Continued)

(22) Filed: **Aug. 1, 2016**

FOREIGN PATENT DOCUMENTS

(65) **Prior Publication Data**  
US 2017/0051874 A1 Feb. 23, 2017

DE 195 33 555 A1 3/1997  
DE 10 2011 078 608 A1 1/2013

(Continued)

(30) **Foreign Application Priority Data**

*Primary Examiner* — John F Pettitt  
(74) *Attorney, Agent, or Firm* — Benoit & Côté Inc.

Aug. 20, 2015 (DE) ..... 10 2015 215 919

(57) **ABSTRACT**

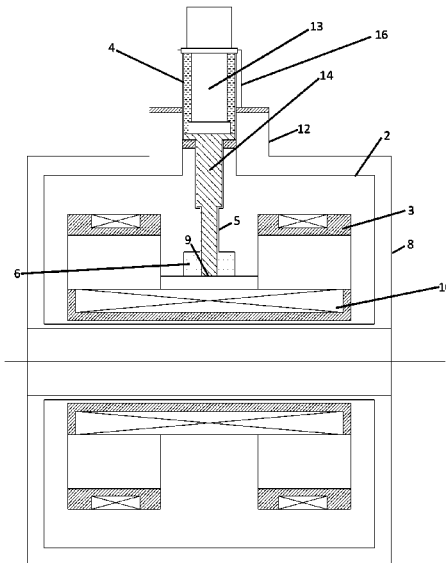
(51) **Int. Cl.**  
**F17C 3/08** (2006.01)  
**F25D 19/00** (2006.01)  
**H01F 6/04** (2006.01)  
**F25B 9/10** (2006.01)  
**F17C 13/00** (2006.01)  
**G01R 33/38** (2006.01)  
**F25B 9/14** (2006.01)

A method is provided for precooling a cryostat having a hollow cold head turret into which a neck tube protrudes and connects an object to be cooled to the exterior of the cryostat, wherein a cold head having a cold head stage for cooling a cryogenic working medium may be introduced into the neck tube. During a condensation operation the cryogenic working medium flows through a heat pipe into an evaporator chamber which is thermally conductively connected to the object to be cooled. During a precooling phase a precisely fitting, thermally conductive short circuit block is inserted through the neck tube into the heat pipe to provide thermal conduction between the object to be cooled and a cooling device. The short circuit block is removed from the heat pipe after the target temperature is reached, and heat is subsequently transmitted through the heat pipe during a condensation operation.

(52) **U.S. Cl.**  
CPC ..... **F17C 3/085** (2013.01); **F17C 3/08** (2013.01); **F17C 13/007** (2013.01); **F25B 9/10** (2013.01); **F25D 19/006** (2013.01); **G01R 33/3804** (2013.01); **H01F 6/04** (2013.01); **F25B 9/14** (2013.01); **F25B 2400/17** (2013.01)

(58) **Field of Classification Search**  
CPC .... F17C 3/08; F17C 3/085; F17C 3/10; F25D 19/006; H01F 6/04

**6 Claims, 4 Drawing Sheets**



(56)

**References Cited**

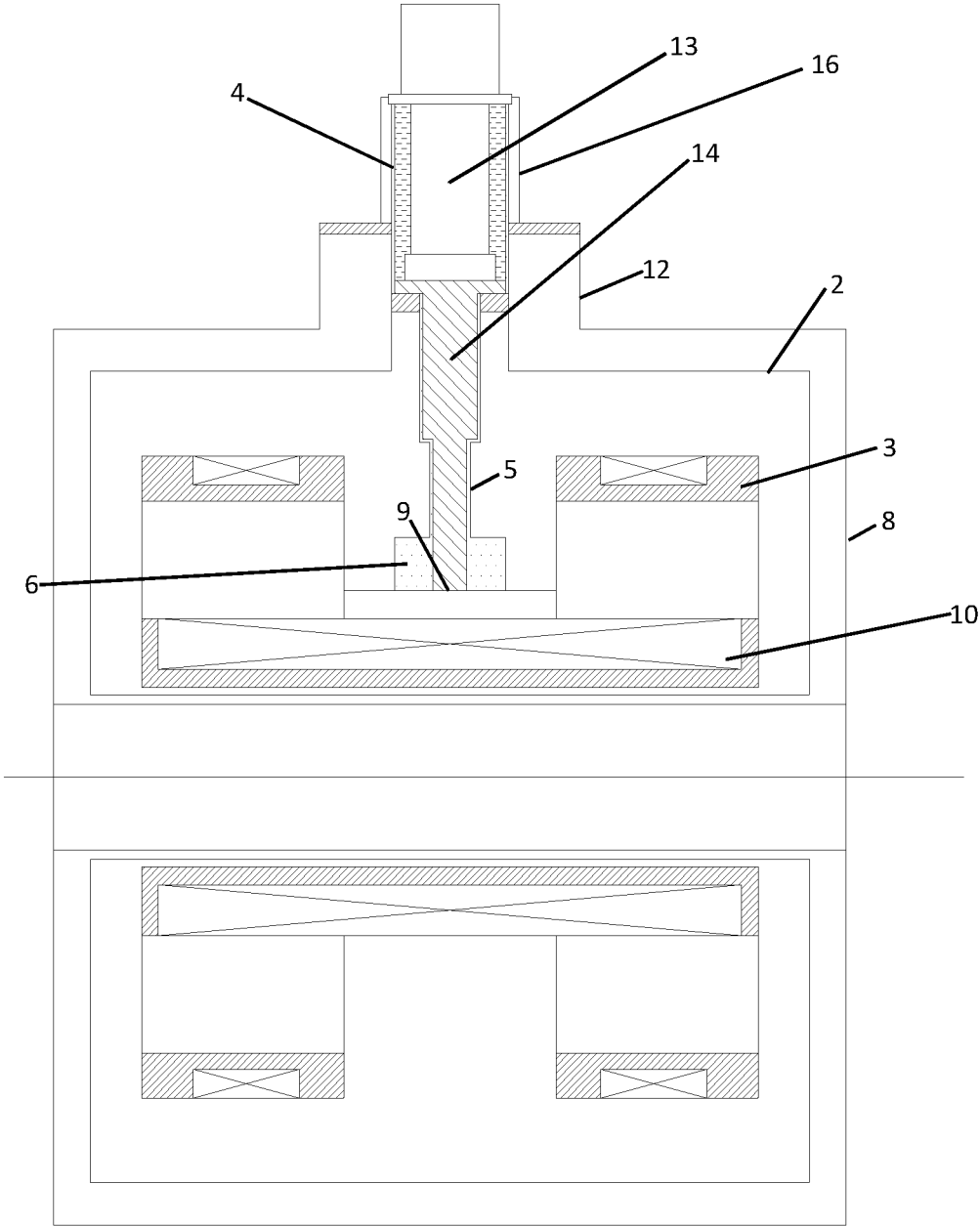
U.S. PATENT DOCUMENTS

2004/0060303 A1\* 4/2004 Habermusch ..... F02G 1/043  
62/6  
2007/0271933 A1\* 11/2007 Miki ..... C01B 3/0089  
62/51.1  
2011/0179808 A1\* 7/2011 Kagan ..... F25D 19/00  
62/48.2  
2012/0102975 A1 5/2012 Garside et al.  
2013/0008187 A1 1/2013 Kraus et al.  
2015/0196221 A1\* 7/2015 Garside ..... F25B 9/14  
600/410  
2016/0084440 A1 3/2016 Strobel

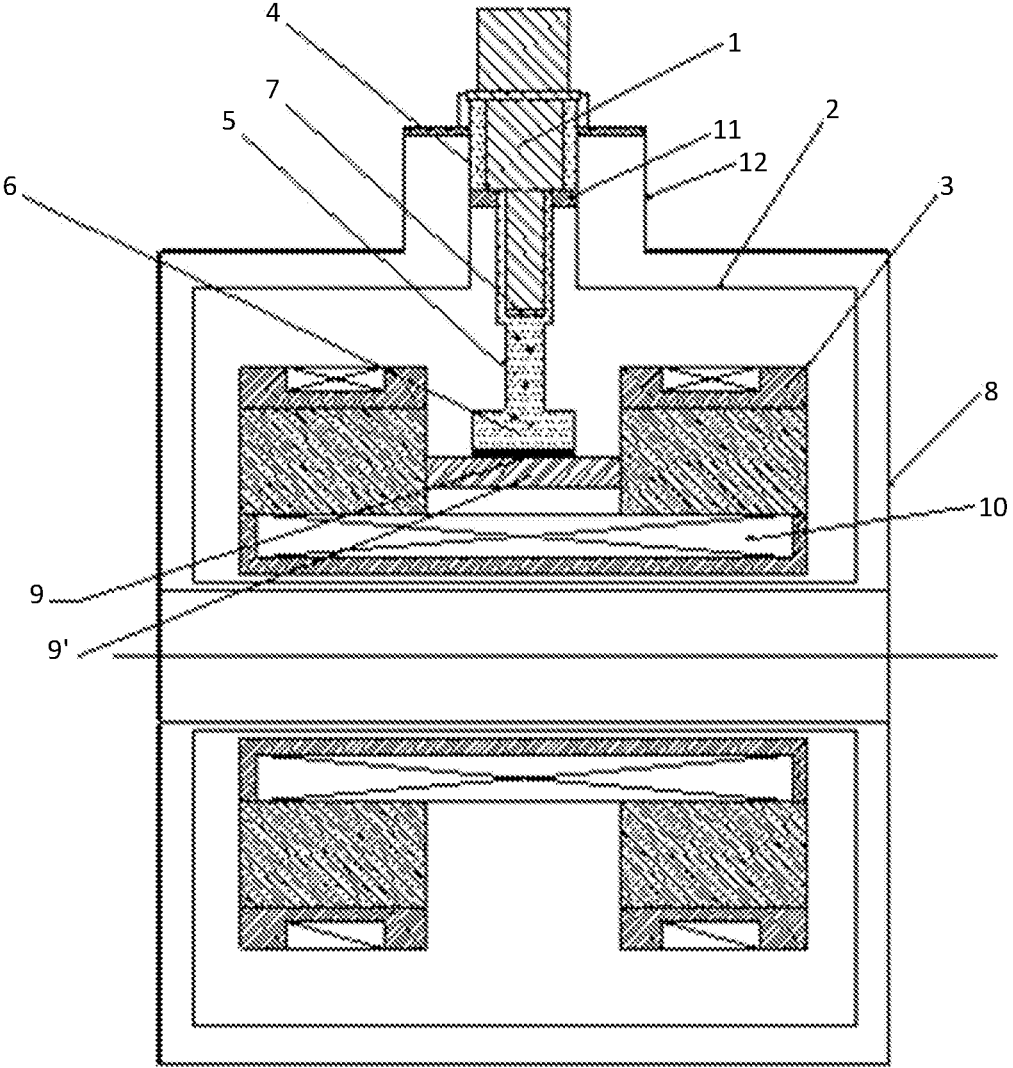
FOREIGN PATENT DOCUMENTS

DE 10 2014 218 773 A1 3/2016  
EP 0359262 A2 3/1990

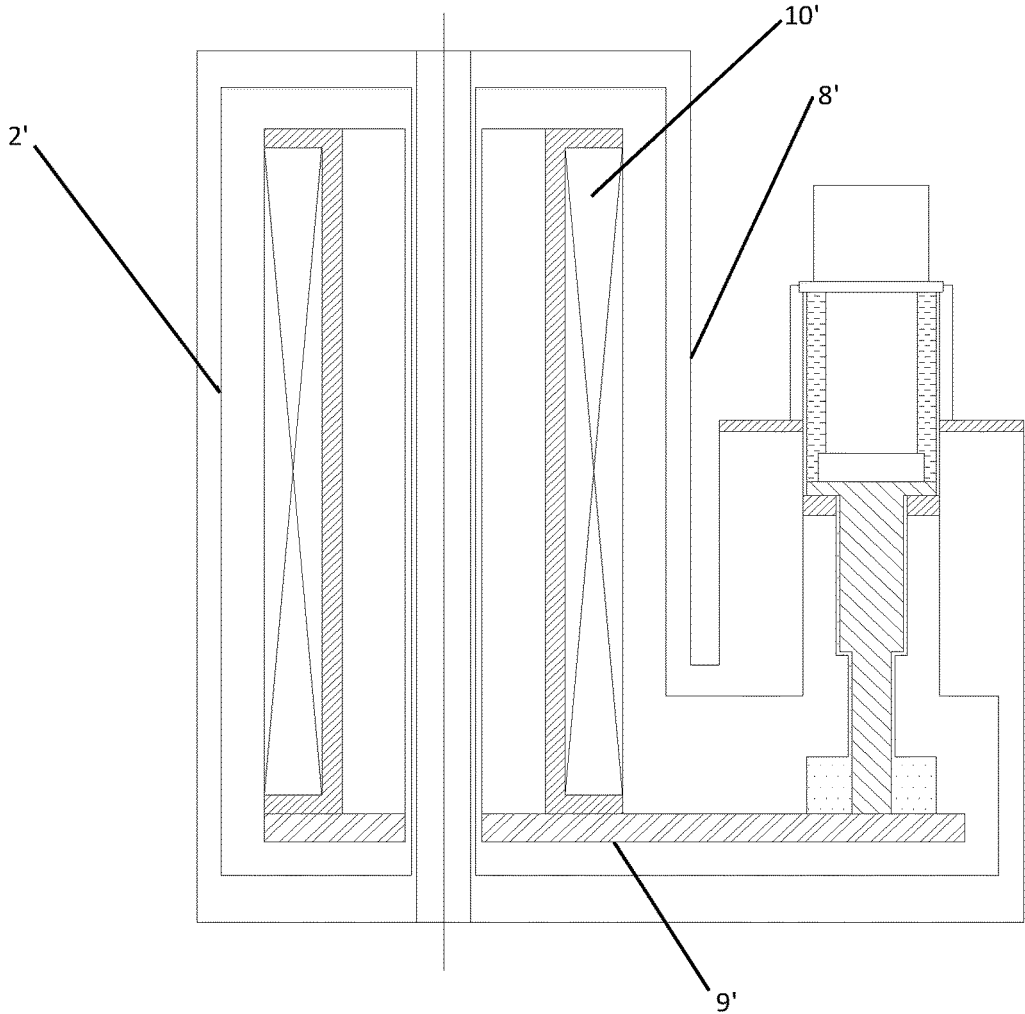
\* cited by examiner



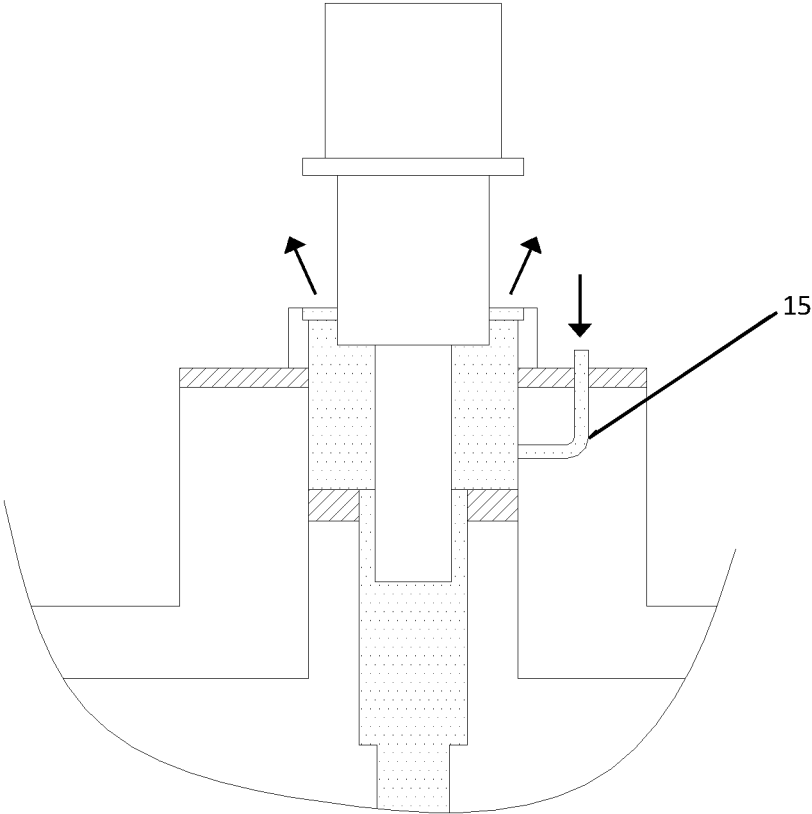
**Fig. 1a**



**Fig. 1b**



**Fig. 2a**



**Fig. 2b**

## METHOD AND DEVICE FOR PRECOOLING A CRYOSTAT

### BACKGROUND OF THE INVENTION

#### Field of the Invention

The invention relates to a method for operating a cryostat which houses a chamber containing at least one object to be cooled.

#### Description of the Related Art

A cryostat is described in patent application DE 10 2014 218 773, unpublished at the time of filing of the present invention, that comprises a vacuum container having at least one hollow cold head turret which contains a neck tube connecting a chamber containing at least one object to be cooled through the outer shell of the vacuum container to the area outside of the cryostat, the neck tube being geometrically configured in such a way that a cold head may be inserted, the cold head having a cold head stage via which a cryogenic working medium, which flows or drips through a heat pipe into an evaporator chamber during a condensation operation, may be cooled, and which is thermally conductively connected to the object to be cooled via a thermal contact surface, so that the cooled cryogenic working medium is able to absorb heat from the object to be cooled and transport it to the cold head stage via the heat pipe.

In most cases, cryotechnology utilizes cooling machines for cooling objects, for example superconducting magnet coils. The cooling machines discharge heat from the apparatus containing the object to be cooled by means of a cold head.

These cooling machines are typically operated with helium gas as the coolant, which is compressed in a compressor and expanded in the cold head of the cryostat (for example, so-called pulse tube coolers). The cold head and the compressor are generally connected to one another via two pressure lines. The cold head is connected to the components to be cooled, either directly mechanically or via a contact medium (for example, cryo gas or cryo liquid), or in both ways in order to ensure good heat transfer.

However, if the compressor completely or partially fails, for example due to a technical defect or power outage, the previously cooled components heat up. In this situation, the cold head of the cryostat then represents a significant thermal bridge between the components to be cooled and the external surroundings. In the persistent operating mode of a superconducting magnet, the superconducting current can flow practically without resistance for extremely long time periods. However, heating of the magnet causes a so-called quench of the persistent operating mode after a certain time. At some point, the magnet reaches the critical transition temperature, which is predetermined by the superconducting material, and becomes normally conducting and thereby loses its high magnetic field, generally abruptly.

A reduction of the thermal load after failure of the cooling machine would at least considerably extend the time period until a quench occurs. This is true in particular for cryostat configurations that can be operated completely without or with only minimum amounts of liquid coolant; superconducting magnets currently are normally operated in a liquid helium bath. Since helium is becoming increasingly expensive, cryostats that can be operated completely without or at least with minimum amounts of helium (low-loss or even cryo-free systems) are becoming more and more attractive, both technically and economically.

However, the thermal capacity of solids significantly decreases at very low temperatures. For this reason, it would be particularly important for systems of this type, using small amounts of liquid helium or no liquid helium at all, to minimize the heat input into the object to be cooled in case of failure of the cooling unit.

DE 195 33 555 A1 describes a cooling device for indirectly cooling a unit, in particular a superconducting unit, situated in a vacuum housing, and which contains at least one refrigerator portion. This refrigerator portion is composed of a section on the room temperature side and a section on the low-temperature side, protrudes into the vacuum housing, and is fastened thereto via spring elements, and at its end on the low-temperature side is thermally conductively connected to the unit to be cooled. To avoid vibrations being transmitted to the unit, it is provided that the section of the cooling machine portion on the room temperature side is situated in an evacuable space of a housing unit which is fixedly connected to the vacuum housing. This cooling device is thermally conductively connected to the unit to be cooled, so that in the event of failure of the cooling machine, the object to be cooled heats up quickly. When an LTS magnet is used, this means that a quench occurs immediately.

LTS coils are conventionally kept directly in liquid helium, which is very costly due to the fact that helium bath cryostats are very complex and therefore expensive.

DE 10 2011 078 608 A1 discloses a cryostat system comprising a vacuum container and a cryo container installed therein, and a sleeve in which a cryocooler is installed, the upper, warm end of the sleeve being connected to the outer shell, and the lower, cold end facing the cryo container being sealed gas-tight by a sleeve base, and the cryo container containing a superconducting magnet system. The known system is characterized in that the cryo container is hermetically sealed except for a gas capillary, and is filled with gaseous fluid at a pressure below the vapor pressure of the liquid phase of the fluid at the corresponding operating temperature, and the coldest stage of the cryocooler is in good thermally conductive connection with a heat exchanger situated inside the cryo container. In this way, the superconducting magnet system within the cryo container can be cooled without cryogenic liquid, and at the same time, without a direct mechanical coupling to the cryocooler; during the operating period, handling of cryogenic liquids may be dispensed with, and the discharge of cold fluid in the event of a quench of the superconducting magnet system may be avoided.

DE 10 2011 078 608 A1 describes a cryostat without cryogen and without a heat pipe. One disadvantage, however, is that if the cooling unit fails, the user has very little time ("time to quench") until the magnet heats up and loses its superconducting property. In contrast, a heat pipe can better bridge over a fairly long failure in the cooling, since the cryogen in the heat pipe evaporates first before the cryostat begins to heat up.

The advantages of a cryostat with a heat pipe with regard to the time to quench are described in the patent application DE 10 2014 218 773 cited above. This document describes a cryostat which includes a vacuum container and a chamber containing the object to be cooled, and includes a neck tube which houses a cooling arm of a cold head which is connected to a refrigeration device, and which can be brought into thermal contact with a second thermal contact surface on the object via a first thermal contact surface on the cooling arm. The described system is characterized in that the hollow volume between the inner side of the neck

tube, the cooling arm, and the object is filled with gas, wherein the internal pressure of the gas pressurizes part of the cooling arm, whereas another part of the cooling arm is pressurized by atmospheric pressure, that the cooling arm can be moved with its first thermal contact surface toward or away from the second thermal contact surface, and that a contact device establishes thermal contact of the first thermal contact surface with the second thermal contact surface when the gas pressure is below a threshold pressure, and the contact device moves the first thermal contact surface away from the second thermal contact surface when the threshold pressure is exceeded, so that the contact surfaces are thermally separated by a gap filled with gas. In this way, in the event of failure of the cooling machine, the thermal load on the object to be cooled is significantly reduced automatically in an operationally reliable manner without intervention by an operator.

However, this type of cryostat requires a relatively long precooling period before it is actually started up. Heat pipes, despite their other advantages, basically are not powerful enough to quickly cool cryostats down to their working temperature, in particular in the event of failure of the cooling unit, described above. For fairly large cryostats containing, for example, an LTS coil weighing several hundred kilograms, a standard heat pipe with 40 W cooling power, with use of a single cryogen, would require several weeks to precool the coil to a working temperature of 4 K.

Even with successive use of different cryogens having different condensation temperatures in the various subareas, within the fairly large temperature difference from the original room temperature down to the very low operating temperature, a precooling time in the range of week is still required; however, this procedure is naturally extremely complicated, in particular material- and labor-intensive.

#### SUMMARY OF THE INVENTION

For a cryostat of the type described at the outset, having a heat pipe, the object of the invention—which, considered in detail, is relatively challenging and complex—is to significantly reduce the time required for precooling the cryostat to its operating temperature, and thus, to shorten the duration of the start-up time, using uncomplicated technical means; existing units may be retrofitted in a preferably simple manner.

This object is achieved surprisingly easily and effectively by the present invention, in that, during a precooling phase, the object to be cooled is precooled to a target temperature of the working range of the cryogenic working medium in which the heat pipe is able to operate efficiently, wherein for the precooling, a precisely fitting short circuit block with good thermal conductivity is inserted through the neck tube into the heat pipe, one free end of the short circuit block being thermally connected to a high-power cooling device, and its other end contacting the thermal contact surface, and that in an intermediate phase the short circuit block is removed from the heat pipe after the target temperature is reached, and that during an operating phase, heat is subsequently transmitted through the heat pipe during a condensation operation.

The concept underlying the present invention is based, among other things, on establishing a thermal short circuit while the cryostat (generally containing a magnet) is being cooled down, until the efficient working range of the heat pipe is reached. A heat pipe is a tube in which the working gas is cooled and condensed at the upper end, in order to then drip or flow to the lower end, where it is evaporated,

thereby absorbing heat. Heat may thus be transmitted with a minor temperature gradient. Unfortunately, the working range is determined by the boiling point of the gas, and therefore is greatly limited. As mentioned above, without additional measures, the cold head of the pulse tube cooler in combination with the heat pipe is not efficient enough above the working temperature to accomplish this cooling within an adequately short time period. Therefore, according to the teaching of the present invention, the cold head is removed from the neck tube and the connecting sleeve, and a precisely fitting material core with good thermal conductivity (for example, copper, aluminum, etc.) is inserted into the cavity, resulting in the largest possible thermal contact surface with the parts to be cooled. A cooling unit is mounted on the outer side of the core, and cools the core and thus the parts of the cryostat in contact with the core. This cooling process now proceeds much more quickly, since high-power cooling units (for example, a Gifford-McMahon cooler with up to 1200 W) may be used.

As soon as the temperature has reached the efficient range of the heat pipe in the cryostat, the cooling unit and the material core may be removed and a two-stage cooler (a pulse tube cooler, for example), which provides sufficient power for continuous operation may be inserted. When the equipment is exchanged, however, it must be ensured that no air or moisture enters the cavity. This is achieved by seals and by blowing in gaseous helium, as described in greater detail below.

When helium is used as the cryogenic working medium, in the intermediate phase it is advantageous to cool the cryostat to a target temperature of 40 K-80 K, since in this range the two-stage cooler in combination with the heat pipe can operate efficiently, and the target temperature of 4 K may be reached relatively quickly. The efficiency of the two-stage cooler drops at temperatures higher than 100 K. In turn, the high-power one-stage cooler loses efficiency at very low temperatures of less than 40 K.

In addition to the cooling process, the invention also proposes a corresponding mobile precooling device with which the method according to the invention may be carried out. The cooling device includes the precisely fitting material core with good thermal conductivity and the high-power cooling unit, which is operated with the same compressor as the pulse tube cooler of the cold head in the heat pipe (connecting sleeve). The connecting sleeve requires a connection for the cryogen supply, so that when a change is made from the precooler to the pulse tube cooler, the resulting cavity is completely flooded with the working cryogen, generally helium.

In a preferred variant of the method according to the invention the cryogenic working medium is blown into the inner space of the neck tube during the removal of the short circuit block from the neck tube in the intermediate phase, and during insertion/placement of the cooling device or the cold head through the neck tube at the start of the operating phase, so that the heat pipe is loaded with the cryogenic working medium. This prevents air from penetrating into the neck tube area and freezing at that location. The contact surfaces must be free of ice to ensure optimal heat transfer.

In another advantageous method variant of the invention, the cryogenic working medium is selected from the following group: helium, neon, nitrogen, hydrogen, or carbon dioxide, depending on the temperature range to be reached in the operating phase. It is thus possible to achieve various working temperatures in different temperature ranges.

Also particularly preferred are variants of the method according to the invention in which it is provided that in the

5

intermediate phase, the high-power cooling device having the short circuit block is exchanged with a two-stage cryo-cooler having a cold head stage.

The present invention further relates to a cryostat for carrying out the method according to the invention described above in detail, which is characterized in that a gas supply device for blowing the cryogenic working medium into the common inner space of the neck tube, heat pipe, and evaporator chamber is present, so that a gas flow in the direction of the neck tube may be provided which prevents air and/or moisture from penetrating into the cryostat during the removal of the short circuit block from the neck tube during the intermediate phase, and during insertion of the heat pipe through the neck tube at the start of the operating phase, and which at the same time loads the heat pipe. This prevents air from penetrating into the neck tube area and freezing at that location. The contact surfaces must be free of ice to ensure optimal heat transfer.

In one advantageous embodiment of the cryostat according to the invention, a sealing device is present which prevents gas exchange from the neck tube to the ambient air.

In one embodiment of the invention which is particularly simple but important, the short circuit block is manufactured so that its outer circumference makes a precise geometric fit with the outer circumference of the heat pipe. Optimal use is thus made of the cross section of the heat pipe in order to transfer the largest possible quantity of heat.

Further preferred embodiments of the cryostat according to the invention provide that the short circuit block is made of a material having a thermal conductivity of  $>200$  W/m·K, in particular copper or aluminum.

Also advantageous are embodiments characterized in that the high-power cooling device is protected from undesirable external heat input by a vacuum device and/or an insulating jacket.

In another embodiment, a tank containing liquid helium is present, with which the thermal contact surface may be cooled with respect to the object to be cooled (generally a superconducting magnet), but the object to be cooled itself is not situated in a tank containing liquid helium. This small tank increases the operational reliability of the magnet in the event of a power outage.

Lastly, in one class of preferred embodiments which have proven to be particularly satisfactory in practice, the high-power cooling device has a cooling power of at least 300 W, preferably at least 500 W, at 80 K.

The present invention further relates to a magnetic resonance system having a superconducting magnet coil, in particular an NMR spectrometer system or an NMR tomography system, but also an MRI or FTMS apparatus, each including a cryostat according to the invention as described above. Due to the present invention, the superconducting magnet coil of the magnetic resonance system is protected particularly well from a quench of the persistent operating mode, and is therefore particularly well suited for high-resolution measurements. A magnetic resonance system of this type typically includes at least one magnet which is generally superconducting and which is situated in a cryostat, as well as high-frequency components, for example HF coils, in a room temperature bore of the cryostat, and a sample position for a sample to be measured. "Normal" conventional high field NMR spectrometers operate at a proton resonance frequency between approximately 200 MHz and 500 MHz. In contrast, a high field NMR spectrometer with ultra-high resolution can be operated nowadays at proton resonance frequencies  $\geq 800$  MHz.

6

Further advantages of the invention result from the description and the drawings. In addition, in each case the features mentioned above and explained in greater detail below may be used according to the invention, individually or collectively in any combination. The embodiments shown and described are not to be construed as an exhaustive listing, but, rather, are exemplary in nature for describing the invention.

## BRIEF DESCRIPTION OF THE DRAWINGS

The invention is illustrated in the drawings and is explained in greater detail with reference to exemplary embodiments, as follows:

FIG. 1a shows a schematic vertical sectional view of one embodiment of a cryostat according to the invention, with a short circuit block during the precooling phase, for example a high-power Gifford-McMahon cooler, in an NMR spectrometer or NMR tomography device having a horizontal magnet system;

FIG. 1b shows a configuration with a heat pipe according to the prior art, similar to the cryostat in FIG. 1a, but without the modifications according to the invention, and having a pulse tube cooler during continuous operation;

FIG. 2a shows a schematic vertical sectional view of another embodiment of the cryostat according to the invention with functional features similar to FIG. 1a during the precooling phase, but in an NMR spectrometer or NMR tomography device having a vertical magnet system; and

FIG. 2b shows a configuration similar to FIG. 2a, but in greater detail and during the intermediate phase after the target temperature is reached, when the short circuit block is removed from the heat pipe, in this case a gas supply device for blowing the cryogenic working medium into the shared inner space of the neck tube, heat pipe, and evaporator chamber being present.

## DETAILED DESCRIPTION

FIGS. 1a, 2a, and 2b each show a schematic vertical sectional view of embodiments of the cryostat according to the invention, while FIG. 1b shows a cooling system with a heat pipe according to the prior art.

All cryostats illustrated in the drawings have a vacuum container which houses a chamber containing at least one object 10; 10' to be cooled, the vacuum container having at least one hollow cold head turret 12 into which a neck tube 4, which connects the chamber through the outer shell 8; 8' of the vacuum container to the area outside of the cryostat, protrudes, the neck tube 4 being geometrically configured in such a way that a cold head 1 may be introduced, the cold head having a (generally, second) cold head stage 7 via which a cryogenic working medium, which flows or drips through a heat pipe 5 into an evaporator chamber 6 during a condensation operation, may be cooled, and which is thermally conductively connected to the object 10; 10' to be cooled via a thermal contact surface 9; 9', so that the cooled cryogenic working medium is able to absorb heat from the object 10; 10' to be cooled and transport it to the cold head stage 7 via the heat pipe 5.

The chamber containing the object 10; 10' to be cooled is surrounded by a radiation shield 2; 2' inside the vacuum container.

A connection 11 having good thermal conductivity is provided between the radiation shield 2; 2' and a first stage of the cold head 1.

The object **10**; **10'** to be cooled is generally a superconducting NMR coil system mounted on or in a coil body **3**.

According to the invention, in the cryostats shown in FIGS. **1a**, **2a**, and **2b**, during a precooling phase the object **10**; **10'** to be cooled is pre-cooled to a target temperature of the working range of the cryogenic working medium in which the heat pipe **5** is able to operate efficiently, wherein for the precooling, a precisely fitting short circuit block **14** with good thermal conductivity is inserted through the neck tube **4** into the heat pipe **5**, one free end of the short circuit block being thermally connected to a high-power cooling device **13**, and its other end contacting the thermal contact surface **9**; **9'**. In an intermediate phase the short circuit block **14** is removed from the heat pipe **5** after the target temperature is reached, and during an operating phase, heat is subsequently transmitted through the heat pipe **5** during a condensation operation.

The cryogenic working medium is blown into the inner space of the neck tube **4** during the removal of the short circuit block **14** from the neck tube **4** in the intermediate phase, and during insertion/placement of the cooling device **13** or the cold head **1** through the neck tube **4** at the start of the operating phase, so that the heat pipe **5** is loaded with the cryogenic working medium.

The cryogenic working medium is selected from helium, neon, nitrogen, hydrogen, or carbon dioxide, depending on the temperature range to be reached in the operating phase.

In the intermediate phase, the high-power cooling device **13** having the short circuit block **14** is replaced by a two-stage cryocooler having a cold head stage **7**.

The cryostat according to the invention is characterized in that a gas supply device **15** for blowing the cryogenic working medium into the shared inner space of the neck tube **4**, heat pipe **5**, and evaporator chamber **6** is present, so that a gas flow in the direction of the neck tube may be provided which prevents air and/or moisture from penetrating into the cryostat during the removal of the short circuit block **14** from the neck tube **4** during the intermediate phase, and during insertion of the heat pipe **5** through the neck tube **4** at the start of the operating phase, and which at the same time loads the heat pipe **5**.

A sealing device is generally present which prevents gas exchange from the heat pipe **5** to the ambient air.

The short circuit block **14** is made of a material having a thermal conductivity of  $>200 \text{ W/m}\cdot\text{K}$ , in particular copper or pure aluminum, so that its outer circumference makes a precise geometric fit with the outer circumference of the heat pipe **5**.

The high-power cooling device **13** has a cooling power of at least 300 W, preferably at least 500 W, at 80 K, and is protected from undesirable external heat input by a vacuum device and/or an insulating jacket **16**, since the cooling device protrudes from the neck tube, and in this area must be shielded from lateral heat input from the surroundings.

In the embodiments of the invention illustrated in the drawings, a tank containing liquid helium is present, with which the thermal contact surface **9**; **9'** may be cooled with respect to the object **10**; **10'** to be cooled, but the object **10**; **10'** to be cooled itself is not situated in a tank containing liquid helium.

In the embodiment shown in FIG. **1a**, the cryostat according to the invention is part of an NMR apparatus having a horizontal magnet system.

In comparison, FIG. **1b** shows a cryostat according to the prior art, as described, for example, in patent application DE 10 2014 218 773, unpublished at the time of filing of the present invention.

The embodiments of the cryostat according to the invention illustrated in FIGS. **2a** and **2b** are in each case part of an NMR apparatus having a vertical magnet system.

FIG. **2b** shows one embodiment of the invention in which a gas supply device **15** for the cryogen is provided when the short circuit block **14** (FIG. **1a**) having the high-power cooling device **13** is exchanged with a pulse tube cooler. The direction of flow of the cryogen during the exchange is indicated by arrows.

In one advantageous design, the high-power cooling device **13** is a Stirling cooler, not a Gifford-McMahon cooler. These coolers are even more efficient, but require a dedicated compressor and are more costly.

The high-power cooling device **13** and the pulse tube cooler are preferably operated with the same compressor unit.

#### LIST OF REFERENCE NUMERALS:

- 1** Cold head
- 2**; **2'** Radiation shield
- 3** Coil body
- 4** Neck tube
- 5** Heat pipe
- 6** Evaporator chamber
- 7** Cold head stage (second stage)
- 8**; **8'** Outer shell
- 9**; **9'** Thermal contact surface
- 10**; **10'** Object to be cooled
- 11** Radiation shield-cold head connection (first stage)
- 12** Cold head turret
- 13** Cooling device
- 14** Short circuit block
- 15** Gas supply device
- 16** vacuum device and/or insulating jacket

The invention claimed is:

**1.** A cryostat that maintains an object to be cooled at a target temperature during an operating phase, and that reduces a temperature of the object to be cooled to the target temperature during a precooling phase, the cryostat comprising:

a vacuum container which houses a chamber containing said object to be cooled, the vacuum container having at least one hollow cold head turret which includes a neck tube that connects the chamber through an outer shell of the vacuum container to an area outside of the cryostat;

a two-stage cooler having a cold head and a cold head stage configured to reside in the neck tube for the operating phase, during which the cold head stage cools a cryogenic working medium, which condenses and flows or drips through a heat pipe into an evaporator chamber, and which cold head stage is thermally conductively connected to the object to be cooled via a thermal contact surface, so that the cooled cryogenic working medium is able to absorb heat from the object to be cooled and transport the heat to the cold head stage via the heat pipe;

a high-power cooling device with a cooling power of at least 300W at 80K configured to be inserted into the neck tube;

a short circuit block that is configured to be inserted through the neck tube into the heat pipe prior to the operating phase, and that resides therein during the precooling phase while the object to be cooled is pre-cooled to the target temperature, the short circuit block having an outer circumference that fits within a

circumference of the heat pipe with one free end external to the heat pipe and thermally connected to the high-power cooling device, and the other end of the short circuit block contacting the thermal contact surface, the short circuit block thereby operating as a material core to facilitate cooling of the thermal contact surface by the high-power cooling device; and

a gas supply device for blowing the cryogenic working medium into a shared inner space of the neck tube, heat pipe, and evaporator chamber during removal of the short circuit block following the precooling phase and insertion of the two-stage cooler for the operating phase, the gas supply device configured to prevent air and/or moisture from entering the cryostat while providing the working medium to the heat pipe.

2. The cryostat according to claim 1, further comprising a sealing device which prevents gas exchange from the heat pipe to ambient air.

3. The cryostat according to claim 1, wherein the short circuit block is made of a material having a thermal conductivity of  $>200 \text{ W/m}\cdot\text{K}$ .

4. The cryostat according to claim 1, wherein the cooling device is protected from external heat input by at least one of a vacuum device and an insulating jacket.

5. The cryostat according to claim 1, wherein the object to be cooled is not in contact with the working medium.

6. The cryostat according to claim 1, wherein the cooling device has a cooling power of at least 500 W at 80 K.

\* \* \* \* \*