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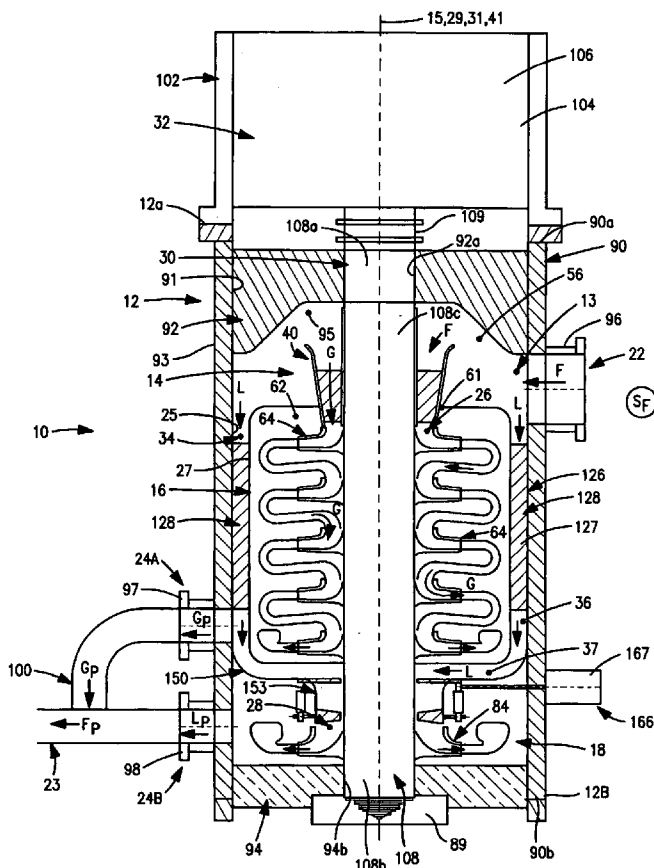
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(54) Title: MULTIPHASE FLUID PROCESSING DEVICE

(57) Abstract: A fluid processing machine processes multiphase fluid streams including gas and liquid. A housing has an interior chamber, an inlet fluidly connected with the interior chamber and with a stream source, and first and second outlets. A separator disposed within the housing chamber is fluidly coupled with the inlet such that the stream flows thereto and separates the stream into gaseous and liquid portions. A compressor disposed within the chamber receives and compresses the gaseous portions from the separator for discharge through the housing first outlet, the compressor having an outer surface spaced from the housing inner surface to define a flow passage. A pump disposed within the chamber has an inlet fluidly coupled with the separator through the passage, is spaced vertically from the separator so that liquid flows by gravity from the separator to the pump, and pressurizes the liquid for discharge through the housing second outlet.



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TITLE OF THE INVENTION
Multiphase Fluid Processing Device

The present invention relates to fluid machinery, and more specifically to
5 machinery for handling multiphase fluid streams.

A variety of devices for handling fluid streams, such as separators, compressors
and pumps, are known. A separator basically functions to separate a fluid stream into
different phases, such as into liquid and gaseous portions, and/or may be used to remove
solid matter from a fluid stream. Compressors and pumps basically function to
10 respectively compress gases and pressurize liquids, often for the purpose of transporting a
fluid. Typically, when a fluid stream is composed of both gaseous and liquid portions, the
fluid stream is first separated, and then the gaseous portions are directed into a compressor
and the liquid portions are directed into a pump so as to be separately treated.

15 SUMMARY OF THE INVENTION

In one aspect, the present invention is a fluid processing machine for processing a
multiphase fluid stream including a mixture of at least a gas and a liquid. The fluid
processing machine comprises a housing having an interior chamber, an inlet fluidly
connected with the interior chamber and fluidly connectable with a source of the
20 multiphase stream, and first and second outlets. A separator is disposed within the
housing chamber, is fluidly coupled with the inlet such that the fluid stream flows
generally to the separator, and is configured to generally separate the fluid stream into a
substantially gaseous portion and a substantially liquid portion. A compressor is disposed
within the housing chamber and is fluidly coupled with the separator such that the gaseous
25 portion flows into the compressor, the compressor being configured to compress the
gaseous portion and to discharge the compressed gaseous portion through the housing first
outlet. Further, a pump is disposed within the housing chamber and has an inlet fluidly
coupled with the separator, the pump being spaced generally vertically from the separator
such that liquid portions flow at least partially by gravity from the separator to the pump
30 inlet. The pump is configured to pressurize the liquid portions and to discharge the
pressurized liquid portions through the housing second outlet.

In another aspect, the present invention is again a fluid processing machine for
processing a multiphase fluid stream including at least gaseous and liquid fluid portions.
The fluid processing machine comprises a housing including an interior chamber, a central

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axis extending through the chamber, an inner surface extending circumferentially about the axis and at least partially defining the chamber, an inlet fluidly connected with the housing chamber and configured to receive the multiphase stream, and first and second outlets fluidly connected with the chamber. A separator is disposed within the housing chamber, is fluidly coupled with the inlet such that the fluid stream flows generally from the inlet to the separator, and is configured to generally separate the fluid stream into a substantially gaseous portion and a substantially liquid portion. A compressor is disposed within the housing chamber so as to be spaced axially from the separator, is fluidly coupled with the separator so as to receive the gaseous portion from the separator and has an outlet fluidly connected with the housing first outlet. The compressor is configured to compress the gaseous portion and to discharge the compressed gaseous portion through the housing first outlet, the compressor having an outer surface spaced inwardly from the housing inner surface so as to define at least one flow passage extending generally parallel with the housing axis. Further, a pump is disposed within the housing chamber so as to be spaced axially from the separator such that the compressor is disposed generally between the separator and the pump. The pump has an inlet fluidly coupled with the separator through the at least one flow passage so as to receive the liquid portion from the separator, the pump being configured to pressurize the liquid portions and to discharge the pressurized liquid portion through the housing second outlet.

In a further aspect, the present invention is also a fluid processing machine for processing a multiphase fluid stream with gaseous portion and liquid portions. The fluid processing machine comprises a housing including an interior chamber, an inner circumferential surface at least partially defining the chamber, an inlet extending into the chamber and configured to receive the multiphase stream, and two outlets extending out of the chamber. A separator is disposed within the housing chamber, is fluidly coupled with the inlet such that the fluid stream flows to the separator, and is configured to generally separate the fluid stream into substantially gaseous portion and a substantially liquid portion. A compressor is disposed within the housing chamber generally beneath the separator, has an inlet fluidly coupled with the separator so as to receive the gaseous flow from the separator, and is configured to compress the gaseous flow and to discharge the compressed gaseous flow from the housing through the first outlet. The compressor further has an outer surface spaced inwardly from the housing inner surface such that a generally annular flow channel is defined between the two surfaces, the flow channel being fluidly coupled with the separator. Also, a collector chamber section is defined within the housing

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chamber generally beneath the separator, the collector chamber section being fluidly coupled with the annular flow channel and with the housing second outlet and configured to contain a quantity of liquid.

In yet another aspect, the present invention is once against a multiphase fluid processing machine. The fluid processing machine comprises a housing having an interior chamber, an inlet fluidly connected with the interior chamber and fluidly connectable with a source of the multiphase stream, and first and second outlets. A separator is disposed within the housing chamber, is fluidly coupled with the inlet such that the fluid stream flows generally to the separator, and is configured to generally separate the fluid stream into a substantially gaseous portion and a substantially liquid portion. A compressor is disposed within the housing chamber has a casing with an interior chamber and an inlet extending into the chamber, and at least one impeller disposed within the casing chamber. The impeller has an inlet and is configured to compress the gaseous portion, the compressor being configured to discharge the compressed gaseous portion through the housing first outlet. The separator is disposed within the compressor casing inlet such that the gas portion flows directly from the separator into the compressor casing. Further, a pump is disposed within the housing chamber, has an inlet fluidly coupled with the separator so as to receive the liquid portion, and is configured to pressurize the liquid portions and to discharge the pressurized liquid portions through the housing second outlet.

BRIEF DESCRIPTION OF THE SEVERAL VIEWS OF THE DRAWINGS

The foregoing summary, as well as the detailed description of the preferred embodiments of the present invention, will be better understood when read in conjunction with the appended drawings. For the purpose of illustrating the invention, there is shown in the drawings, which are diagrammatic, embodiments that are presently preferred. It should be understood, however, that the present invention is not limited to the precise arrangements and instrumentalities shown. In the drawings:

Fig. 1 is an axial cross-sectional view of a multiphase fluid processing machine in accordance with the present invention;

Fig. 2 is an enlarged, broken-away view of an upper portion of the fluid processing machine, showing a separator, a plenum chamber and a compressor inlet of the fluid processing machine;

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Fig. 3 is a radial cross-sectional view of the fluid processing machine taken through line 4-4 of Fig. 2;

Fig. 4 is a greatly enlarged, top perspective view of the separator;

Fig. 5 is a greatly enlarged, axial cross-sectional view of a separator;

5 Fig. 6 is an enlarged, broken-away view of a central portion of the fluid processing machine, showing a compressor and liquid flow passage of the fluid processing machine;

Fig. 7 is a radial cross-sectional view of the fluid processing machine taken through line 7-7 of Fig. 6;

Fig. 8 is a greatly enlarged, top perspective view of an impeller of the compressor;

10 Fig. 9 is an enlarged, broken-away view of a lower portion of the fluid processing machine, showing a pump and an adjustable vane assembly of the fluid processing machine;

Fig. 10 is a radial cross-sectional view of the fluid processing machine taken through line 10-10 of Fig. 9;

15 Fig. 11 is a radial cross-sectional view of the fluid processing machine taken through line 11-11 of Fig. 9;

Fig. 12 is a top plan view of a baffle plate for distributing liquid flow into the pump; and

20 Fig. 13 is an axial cross-sectional view of an alternative multiphase fluid processing machine, in which the pump is replaced by a fluid collector.

DETAILED DESCRIPTION OF THE INVENTION

Certain terminology is used in the following description for convenience only and is not limiting. The words "upper", "upward", "down" and "downward" designate
25 directions in the drawings to which reference is made. The words "inner", "inwardly" and "outer", "outwardly" refer to directions toward and away from, respectively, a designated centerline or a geometric center of an element being described, the particular meaning being readily apparent from the context of the description. Further, as used herein, the word "connected" is intended to include direct connections between two members without
30 any other members interposed therebetween and indirect connections between members in which one or more other members are interposed therebetween. The terminology includes the words specifically mentioned above, derivatives thereof, and words of similar import.

Referring now to the drawings in detail, wherein like numbers are used to indicate like elements throughout, there is shown in Figs. 1-13 a multiphase fluid processing

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machine 10 for processing a multiphase fluid stream F that includes a mixture of at least a gas G and a liquid L. The fluid processing machine 10 basically comprises a housing 12 having an interior chamber 13, and a separator 14, a compressor 16, and a pump 18 or a liquid collector 20 (Fig. 13) each disposed within the housing chamber 13. The housing 12 has an inlet 22 fluidly connected with the interior chamber 13 and fluidly connectable with a source S_F of the multiphase fluid stream F, and first and second outlets 24A, 24B. The separator 14 is fluidly coupled with the housing inlet 22, such that the fluid stream F flows generally to the separator 14, and is configured to generally separate the fluid stream F into a substantially gaseous portion G and a substantially liquid portion L. The compressor 16 is fluidly coupled with the separator 14 such that the stream gaseous portion G flows into the compressor 16, and is configured to compress the gaseous portion G and to discharge the compressed gaseous portion G through the housing first or "gas" outlet 24A. Preferably, the compressor 16 is disposed generally beneath the separator 14, and is most preferably located generally between the separator 14 and the pump 18. However, the compressor 16 may alternatively be spaced generally horizontally from (i.e., as opposed to vertically lower or beneath), or even generally above the separator 14 such that the separator 14 is disposed generally between compressor 16 and the pump 18.

Further, the pump 18 has an inlet 28 fluidly coupled with the separator 14, and is preferably spaced generally vertically from, so as to be located generally beneath, the separator 14 such that the stream liquid portion L flows at least partially by gravity from the separator 14 to the pump inlet 28. However, the separator 14 and/or pump 18 may be configured such that the stream liquid portion L flows substantially by suction generated by the pump 18, particularly when the separator 14 and pump 18 are horizontally spaced, or by any other appropriate means. In any case, the pump 18 is configured to pressurize the liquid portion L of the flow stream F and to discharge the pressurized liquid portion L through the housing second or "liquid" outlet 24B. Alternatively, the fluid machine 10 may instead have a liquid collector 20 disposed generally beneath the compressor 16 and fluidly coupled with the separator 14 and with the housing second outlet 24B, the collector 20 having chamber 21 configured to contain a quantity or "accumulated volume" VL of the liquid portions L, as shown in Fig. 13.

As best shown in Fig. 1, the fluid processing machine 10 preferably further comprises a drive shaft 30 extending generally vertically through the housing chamber 13 and rotatable about a central axis 31. Each one of the separator 14, the compressor 16, and the pump 18 having at least one rotatable member 40, 64, and 84, respectively, connected

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with the shaft 30 and spaced apart vertically along the shaft axis 31. As such, rotation of the drive shaft 30 about the axis 31 generally operates each one of the separator 14, the compressor 16 and the pump 18, as described in greater detail below. Alternatively, the fluid processing machine 10 may include two or more separate shafts (structure not shown) each rotatably driving one or more of the separator 14, the compressor 16, and the pump 18. Furthermore, the processing machine 10 preferably further comprises a motor 32 connected with the shaft 30 and configured to rotate the shaft 30 about the central axis 31, the motor 32 preferably being mounted to one end 12a or 12b of the housing 12, as discussed in further detail below.

10 Referring to Figs. 1, 6 and 7, the housing 12 preferably includes an inner surface 25 at least partially defining the housing chamber 13 and the compressor 16 has an outer surface 27 spaced from the housing inner surface 25. As such, at least one liquid flow passage 34 is at least partially defined between the two surfaces 25, 27, the flow passage 34 being configured to fluidly couple the separator 14 with the pump 18. Preferably, the housing 12 further has a centerline or axis 29 and the housing inner surface 25 extends circumferentially about the centerline 29, the shaft axis 31 preferably being generally collinear with the housing axis 29 when the drive shaft 30 is disposed within the housing 12. The compressor 16 preferably has a centerline 15 which is collinear with the shaft axis 31, the compressor outer surface 27 extending circumferentially about the centerline 15 and being spaced radially inwardly from, so as to be disposed coaxially within, the casing inner surface 25. As such, a first portion 36 of the liquid flow passage 34 is generally annular and extends both generally along and generally about the housing axis 29. Preferably, the liquid passage annular portion 36 has an annular inlet port 36a fluidly coupled with a plenum chamber 56 (described below) defined about the separator 14 and an annular outlet port 36b fluidly coupled with the pump inlet 28. Specifically, the liquid flow passage 34 preferably further includes a generally radially extending portion 37 defined between the compressor 16 and the pump 18 and directly fluidly coupled with the pump inlet 28. As such, the liquid passage radial portion 37 fluidly connects the passage annular portion 36, and thus the plenum chamber 56 and separator 14, with the pump inlet 28.

30 With the preferred arrangement of the separator 14, the compressor 16 and the pump 18, the annular portion 36 of the liquid flow passage 34 extends generally vertically such that gravity at least partially initiates flow of the liquid through the flow passage 34, as discussed above. More specifically, the separator 14 is configured to direct liquid

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generally radially outwardly toward the housing inner surface 25, such that liquid portions L flow into the liquid flow channel 34 and thereafter flow at least partially by gravity to the pump inlet 28, such flow preferably being further initiated or "driven" by suction at the pump inlet 28. Further, the radial portion 37 of the liquid flow passage 34 preferably
5 extends generally horizontally between the vertical annular portion 37 and the pump inlet 28.

Referring to Figs. 1-5, the separator 14 preferably includes a body 40 rotatable about a central axis 41, the separator body 40 having a first and second end 40a, 40b, respectively. The first or "upper" body end 40a has a first or "stream inlet" opening 42
10 fluidly coupled with the housing inlet 22 so as to receive the fluid stream F and the second or "lower" body end 40b has a second or "gas outlet" opening 44 fluidly coupled with the compressor 16. An inner separation surface 46 extends circumferentially about the axis 41 and generally between the body first and second ends 40a, 40b. Further, the separation surface 46 defines a separation chamber 48 and is angled generally radially outwardly in a
15 direction d_1 toward the body first end 40a. With this structure, as the separator body 40 rotates about the axis 41, liquid portions L of the fluid stream F contact the inner separation surface 46 are directed generally radially outwardly away from the body axis 41 and generally axially toward, and beyond, the body first end 40a. In other words, centrifugal force generated by rotation of the separator 14 causes the relatively heavier
20 liquid portions L contacting the rotating inner separation surface 46 to slide upwardly and outwardly the along the angled inner surface 50 until the liquid portions are projected or "slung" from the body upper end 40a in a spiral path toward the housing inner surface 25. As such, the liquid portions L are directed to flow back out through the body first opening 42 while a remainder of the fluid stream F, i.e., the substantially gaseous portions G, flows
25 in the downward direction d_2 through the body second opening 44, and thereafter into the compressor 16.

Further, the separator 14 preferably further includes an outer separation surface 50 extending circumferentially about the body axis 41 and generally between the body first and second ends 40a, 40b. As with the inner surface 46, the outer separation surface 50 is
30 angled generally radially outwardly in the direction d_1 toward the body first end 40a. As such, as the separator body 40 rotates about the axis 41, liquid portions L of the fluid stream F contacting the outer separation surface 50 are directed generally radially outwardly away from the body axis 41 and generally axially toward the body first end 40 so as to be directed generally toward the housing inner surface 25. In other words,

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centrifugal forces cause the relatively heavier liquid portions L contacting the rotating outer separation surface 50 to slide upwardly and outwardly the along angled outer separation surface 50 until being projected/slung from the separator body upper end 40a in a generally spiral path toward the housing inner surface 25. Furthermore, the separator 14 is preferably located within the housing chamber 13 such that the separator body first, upper end 40a, and thus the fluid stream inlet opening 42, is spaced generally vertically upwardly with respect to the housing inlet 22. As such, the fluid stream F tends to contact or flow across the outer separation surface 50 prior to entering the separator 14, which contributes to separation of the liquid portions L from the flow stream F, as discussed in further detail below.

Referring now to Figs. 1 and 2, the housing 13 preferably further includes a generally concave wall 54 disposed within the interior chamber 13 so as to define a plenum chamber 56, and the separator 14 is disposed at least partially within the plenum chamber 56. The housing plenum wall 54 is arranged with respect to the housing inlet 22 such that multiphase fluid stream F is directed to flow radially about the separator axis 41 and/or generally parallel with respect to the axis 41 prior to entering the separator opening 42. As such, the fluid stream F preferably contacts the plenum wall 54 prior to entering the separator 14 such that liquid portions L tend to adhere to the plenum wall 54. The plenum wall 54 extends generally radially across the upper end of the housing chamber 13 and has an upper enclosed end 54a disposed generally on the housing centerline 29 and a lower edge 54b extending circumferential about the centerline 29 and disposed generally against the housing inner surface 25. As such, liquid portions L adhering to the plenum wall 54 tend to flow toward the wall edge 54b and thereafter into and through the preferred liquid flow passage 34, as described above and in further detail below.

Referring now to Figs. 1 and 6-8, the compressor 16 is preferably a centrifugal compressor 59 that basically includes a casing 60 disposed within the housing 12 and defining an interior chamber 62 and at least one rotatable impeller 64 disposed within the casing chamber 62 and having an inlet 65. The casing 60 has an inlet opening 61 configured to receive a portion of the separator 14 and an outlet 63 fluidly coupled with the housing first, gaseous outlet 24A. Preferably, the separator body 40 is disposed within or extends through the compressor casing inlet 61, such that the separator body second end 40b is disposed within the compressor chamber 42, thereby fluidly coupling the separator gas outlet 44 with the impeller inlet 65. More specifically, the separator body second end 40b is preferably located generally adjacent to the impeller inlet 65, such that gaseous

portions G flow out of the separator outlet 44 and substantially directly into the impeller inlet 65. As such, there is no need for any component (e.g., a pipe or tube) for directing the stream gas portion G from the separator 14 into the compressor 16. However, the separator 14 and compressor 16 may be alternatively arranged such that the separator outlet end 40b is disposed within the impeller opening 65, or even spaced a substantial distance from the opening 65. For example, the separator 14 may be disposed entirely externally of the compressor casing 62 with the separator gas outlet 44 being fluidly coupled with the compressor casing inlet opening 61, such that the stream gas portion G flows through the casing inlet opening 61 prior to entering the impeller inlet 65.

Further, the one or more impellers 64 are each configured to compress the gaseous fluid portions G flowing into the impeller inlet 65 and to discharge the gaseous portions G through the casing outlet 63, which thereafter flows through the housing first outlet 24A, as described in further detail below. Most preferably, the centrifugal compressor 59 is a "multistage" centrifugal compressor having a plurality of compression stages 66, each compression stage 66 including one impeller 64 and one fixed flow channel assembly 68 disposed about the impeller 64. Each flow channel assembly 68 includes a diffuser channel 70, a return bend channel 72, a return channel 74 and a guide vane 76. As such, compressed gaseous portions G flowing radially outwardly in a "swirl" from one compression stage 66 flows outwardly through the diffuser channel 70, directed back radially inwardly as it flows through the return bend channel 72 and the return channel 74, and is directed by the guide vane 76 generally axially into an adjacent compressor stage 66, or with the final stage 66, through the casing outlet 63.

Referring to Figs. 1 and 9-12, the pump 18 is preferably a rotary pump, most preferably a centrifugal pump 80 that basically includes a casing 82 and a rotatable impeller 84. The casing 82 is disposeable within the housing 12 and has an interior chamber 86, an axially extending suction nozzle 87 providing the pump inlet 28, and an outlet or discharge nozzle 88 fluidly coupled with the housing second, liquid outlet 24B. The impeller 84 is configured to pressurize liquid portions L flowing into the suction nozzle 81 and to direct the pressurized liquid L_P toward the discharge nozzle 88, and thereafter to the housing second outlet 24B. More specifically, the impeller 84 preferably has a plurality of circumferential spaced, generally radially extending vanes 83 configured to direct liquid radially outwardly from the rotating impeller 84, thereby imparting kinetic energy to the liquid. Also, the casing 82 is further configured to provide a volute chamber 89 defined generally between the impeller 84 and the discharge nozzle 88, which functions

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to decelerate and pressurize liquid flowing from the impeller 84, as is well known to those skilled in the pump and fluid machinery arts.

With the basic structure described above, the fluid processing machine 10 of the present invention basically functions as follows. A multiphase fluid stream F enters the housing 12 through the housing inlet 22 and flows into the plenum chamber 56, swirls about and flows into the separator 14. Liquid portions L are separated from the remaining, substantially gaseous portions G of the fluid stream F, and are directed into the liquid flow passage 34. Generally simultaneously, the gaseous portions G flow into the compressor inlet 26 and are pressurized or compressed such that the gas pressure is incrementally increased as the gas portions G flow through each compressor stage 66. The pressurized gas portions G_p are discharged from the compressor 16 and flow out the housing through the housing gas outlet 24A. Further, the separated liquid portions L entering the liquid flow passage 34 flow by gravity (and suction) through the passage vertical portion 36, and thus around the compressor 16, and then through the passage horizontal portion 37 beneath the compressor 16 and into the pump inlet 28. The preferred centrifugal pump 80 then pressurizes the liquid portions L_p as the portions L_p are accelerated radially outwardly by the impeller 84, and the pressurized liquid portions L_p flow out of the housing through the liquid outlet 24B. Preferably, the pressurized gas and liquid portions G_p , L_p are merged or remixed in a common outlet pipe 23 connected with both of the housing outlets 24A, 24B, such that the pressurized fluid stream F_p is further processed or utilized, but the two pressurized flows G_p , L_p may alternatively remain distinct so as to be thereafter separately processed or utilized.

Having described the basic components and functions above, these and other elements of the fluid processing machine of the present invention are described in greater detail below.

Referring to Figs. 1, 2, 4, 6, 7 and 9-11, the housing 12 preferably includes a generally circular tubular body 90 with opposing axial ends 90a, 90b and two generally circular cylindrical end walls 92, 94 disposed at a separate body end 90a, 90b, respectively. The tubular body 90 has an inner surface 91 providing the housing inner circumferential surface 25, such that the housing chamber 13 is generally cylindrical shaped, and an opposing outer circumferential surface 93. Further, each housing end wall 92, 94 encloses the proximal end 90a, 90b of the tubular body 90 and has a generally circular central shaft opening 92a, 94a configured to receive a portion of the drive shaft 30. The first or upper end wall 92 is preferably relatively thicker than the second, lower end

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wall 94 and includes a generally frustaconical shaped cavity 95. With this structure, the upper end wall 92 provides the plenum wall 54 and the end wall cavity 95 provides the plenum chamber 56, as described above. Alternatively, the plenum chamber 56 may be provided by a separate dish-shaped body (not shown) disposed generally adjacent to the upper end wall 92. Further, the housing lower end wall 94 preferably includes a bearing assembly 89 disposed at least partially within the wall opening 94a and configured to support the drive shaft 30.

Preferably, the housing inlet 22 and each housing outlet 24A, 24B includes a separate generally circular tube or pipe section 96, 97, 98, respectively, extending outwardly from the outer surface 93 of the housing tubular body 90. The inlet pipe 96 is located generally proximal to the body upper end 90a and preferably has a centerline 96a that extends generally perpendicularly to the housing centerline 29 and spaced below the separator fluid inlet 42, such that the fluid stream F tends to "swirl about" the centerline 29, and thus about the separator 14, upon entry into the plenum chamber 56, as described above. Also, the inlet pipe 96 is connected or connectable with the source of the fluid stream (not shown), preferably by one or more pipes (none shown). Furthermore, each of the two outlet pipes 97, 98 are disposed generally adjacent to the housing body lower end 90b and are each generally radially aligned with the outlet of the compressor 16 and pump 18, respectively, such that the second, liquid outlet pipe 97 is spaced generally axially below the first, gas outlet pipe 98, as shown in Fig. 1. Preferably, the two outlet pipes 97, 98 are connected together so as to extend into a single outlet conduit 100, such that the pressurized gaseous and liquid portions G_p , L_p are merged after respective compression and pressurization. However, the two outlet pipes 97, 98 may be separately connected to additional piping (none shown) such that the pressurized fluid and gaseous portions G_p , L_p remain separated for independent use or further separate treatment.

As shown in Fig. 1, the housing 12 also preferably includes a generally circular tubular motor housing 102 having an inner axial end 102a disposed generally against and attached to the tubular body upper end 90a, but may alternatively be mounted to the housing body lower end 90b. The motor housing 102 has an inner bore providing a motor chamber 104 for housing the preferred motor 32, as discussed above. By arranging the motor housing 102 vertically above the remainder of the processing machine components, the chance of liquid intrusion into the motor 32 is minimized. However, the motor chamber 104 may be otherwise provided, such as for example, mounting the housing 102 to the lower end 90b of the tubular housing 90, forming the housing 102 as an upper,

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integral portion of the tubular body 90 defined by spacing the upper end wall 92 axially inwardly, by a separate housing spaced from the tubular body 90, etc. (neither shown), or the motor 32 may even be directly mounted to the housing 12 without any covering or housing.

5 Further, the motor 32 is preferably an electric motor 106 configured to rotate the shaft 30 about its axis 31, but may alternatively be a hydraulic motor, a steam or gas turbine, or any other appropriate motor, engine or "prime mover". The preferred shaft 30 includes an elongated circular bar 108 positioned in the fluid machine 10 so as to extend longitudinally through the housing chamber 13 and having upper and lower portions 108a,
10 108b disposed within the housing end wall shaft openings 92a, 94a. The shaft body 108 has an outer mounting surface 109 configured to accept at least one rotatable component 40, 64 and 84 of the separator 14, the compressor 16 and the pump 18, as discussed above and in further detail below. Further, the shaft 30 is preferably directly driven by the motor 32, as described above, but may alternatively be driven through a gear train, belt drive or
15 other appropriate transmission means (none shown).

Referring to Figs. 1-5, the separator body 40 is preferably constructed as a generally conical drum 110 that includes an inner tubular hub member 112, an outer tubular member 114 disposed coaxially about a portion of the hub member 112, and a plurality of vanes 116 extending between and connecting the inner and outer tubular
20 members 112, 114. The inner hub member 112 is preferably formed as a generally circular tube and has a central bore 113, a portion 108c of the shaft body 108 being disposed within the bore 113 so as to mount the separator 14 to the shaft 30. Referring particularly to Fig. 5, the outer tubular member 114 has an upper, generally circular inlet end 114a with a first diameter D_1 , a lower, generally circular outlet end 114b with a
25 second diameter D_2 . The first diameter D_1 is larger than the second diameter D_2 , such that the tubular member 114 tapers inwardly in a direction d_2 downwardly along the axis 41. As such, the outer tubular member 114 has inner and outer, generally conical surfaces 115, 117 providing the inner and outer separation surfaces 46, 50, respectively. Each conical surface 115, 117 extends circumferentially about the hub member 112, the inner surface
30 115 facing generally toward the outer member upper end 114a and the outer surface 117 facing generally toward the body lower end 114b.

With the above separator structure, the multiphase flow machine 10 provides the function of separating the fluid stream F into at least gaseous and liquid portions G, L in generally the following manner. The fluid stream F flows through the housing inlet 22 so

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as to initially swirl around the plenum chamber 56 and about the separator 14, causing some of the liquid portions L to adhere to the plenum wall 54 and other liquid portions L to adhere to the outer separation surface 50. The liquid portions L on the plenum wall 54 flow by gravity toward the liquid flow passage 34. Rotation of the separator body 40
5 causes the liquid portions L on the outer separation surface 50 to flow radially outwardly therealong until being projected radially outwardly toward the plenum wall 54, and thereafter flow either directly into the passage 34 or from the wall 54 into the passage 34. Further, the remainder of the fluid stream F is drawn into the separator upper opening 42 by suction at the compressor inlet 26, such that remaining liquid portions L contact the
10 inner separation surface 46. As with the outer surface 50, the liquid portions L on the inner surface 46 are caused to flow radially outwardly along the surface 46 until being projected or slung radially outwardly toward the plenum wall 54, to thereafter flow directly or indirectly into the liquid flow passage 34. The substantially gaseous portions G of the fluid stream then flow through the compressor inlet 26.

15 Although the conical drum separator 110 with inner and outer conical separation surfaces 115, 117 is presently preferred, it is within the scope of the present invention to construct the separator 14 in any other appropriate manner that permits the fluid processing machine 10 to function generally as described herein. For example, the separator 14 may be constructed as a rotary separator having only one of the inner and
20 outer angled separation surfaces 46, 50, the other surface 50 or 46 being substantially vertical or otherwise formed (e.g., a conical outer surface and generally circular bore, a generally cylindrical body with a conical bore, etc). Further for example, the separator 14 may be formed as any other appropriate type of fluid separator device, such as a
25 cylindrical cyclonic separator, an inline rotary separator, or a separator tank, and/or may be configured to remove solid matter from the fluid stream or to separate the fluid stream into two or more liquid portions of varying density. The present invention embraces these and all other appropriate constructions of the separator 14, and the scope of the present invention is in no manner limited to any specific type or/and arrangement of the separator
14.

30 Referring to Figs. 1, 6 and 7, the compressor casing 60 includes a generally circular tubular wall 120 and first and second, generally circular, radially extending end walls 124A, 124B disposed at each axial end of the tubular wall 120. The casing tubular wall 120 has an inner and outer circumferential surfaces 121, 122, the inner surface 121 defining the compressor chamber 62. The tubular body outer surface 122 provides the

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compressor outer surface 27 as described above; specifically, the body inner surface 122 is spaced radially inwardly from the housing inner surface 91 so as to thereby form the generally annular liquid passage vertical portion 36, as described above. Further, each casing end wall 124A, 124B includes a central opening 125A, 125B, respectively. The upper end wall opening 125A is configured to receive a portion of the preferred separator 14, and thus provides the compressor inlet 26, and the lower end wall opening 125B is sized to receive a portion of the drive shaft 30, such that the shaft 30 extends at least partially therethrough.

Preferably, the compressor 16 is mounted within the multiphase fluid machine 10 by a plurality of support members 126 spaced circumferentially about the housing axis 29 and configured to mount the compressor 16 coaxially within the housing 12, as discussed above. Each support member 126 preferably includes or is formed as a vertically-extending, elongated strut 128 that extends between the housing wall inner surface 91 and the casing wall outer surface 122 and is configured to connect the casing 60 with the housing 12. Further, each strut 128 is formed of an elongated rectangular bar 127 having an outer vertical edge 127a attached to the housing inner surface 91 and an opposing inner vertical edge 127b attached with the casing outer surface 122, as indicated in Fig. 7. Further, the struts 127 each extend substantially through the entire axial length of the liquid passage vertical portion 36, such that the liquid flow passage 34 is thereby divided into plurality of vertical passage sections 39 (see Fig. 7). Although the struts 128 are presently preferred, the compressor casing 60 may be retained within the housing 12 by any other appropriate means, such as for example, one or more support blocks (none shown) configured to support and space the compressor 16 radially within the housing chamber 13, etc.

Referring now to Figs. 1 and 6-8, as discussed above, the compressor 16 is preferably a multistage centrifugal compressor 59 having a plurality of compression stages 66, for example five stages 66A, 66B, 66C, 66D and 66E as depicted, that each include a rotatable impeller 64 and a fixed flow channel assembly 68. The impellers 64 are mounted to the shaft 30 so as to be spaced apart vertically along the shaft axis 31 and each flow channel assembly 68 is disposed generally between the associated impeller 64 and the casing body inner surface 121. Each impeller 64 basically includes a generally cylindrical hub 130, a generally conical shroud 132 spaced axially from the hub 130 and a plurality of radial blades 134 extending between the hub 130 and shroud 132 and spaced circumferentially apart from each other. Each impeller inlet 65 is generally defined

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between the associated shroud 132 and hub 130 (or shroud 132 and shaft 30) so as to be generally annular, and each impeller 64 further has a plurality of circumferentially spaced outlets 138 defined between the outer radial blade ends 134a. As such, fluid (i.e., the gaseous portions G) flows generally axially into each impeller inlet 65 and is then directed
5 radially outwardly from the shaft axis 31 through the series of outlets 138, such that the pressure of the gas portion G progressively increases as it passes through each compression stage 66. Further, each flow channel assembly 68 is preferably mounted within the casing 10 by assembling the various structural components (not indicated) of the channel assembly 68 about an impeller 64 mounted on the shaft 30 so as to form one
10 compressor stage 66, and is configured to provide one diffuser channel 70, one return bend channel 72, one return channel 74 and a guide vane 76, each such component being generally known in the art of centrifugal compressors.

With the above compressor structure, gaseous portions G flow into the compressor 14 through the casing inlet 61 (i.e., as the gas portion G passes to the separator gas outlet
15 44) and then flows from the separator outlet 44 generally directly into the inlet 65 of the impeller 65 of the first stage 66A. The gaseous portions G are accelerated by the rotating blades 134 to flow both radially and tangentially outwardly from the impeller outlets 138 so as to "swirl" in a generally spiral manner through the associated diffuser channel 70, is curved back inwardly toward the shaft 30 by the connected return bend channel 72, such
20 that swirl is removed in the return channel 74 as the fluid flows back toward the shaft 30, and is diverted by the guide vane 76 to flow axially into the impeller inlet 136 of the adjacent or second compressor stage 66B. This flow pattern is repeated through the second, third, fourth stages 66B, 66C, and 66D, etc. However, the flow channel assembly 68 of the last compressor stage 66E includes only a diffuser channel 95 that directs fluid
25 flow into a volute 140 prior to flowing out of the compressor outlet 63 and to the housing first, gas outlet 24A.

Although a multistage centrifugal compressor 59 as described above is presently preferred, the compressor 16 may alternatively be constructed as a single stage centrifugal compressor (not shown) or any other type of compressor capable of compressing a gas,
30 such as for example, a reciprocating compressor, a rotary screw compressor, etc. It is within the scope of the present invention to construct the compressor 16 in these or any other appropriate configuration that is capable of compressing the gaseous portions G in a manner such that the multiphase fluid machine 10 is capable of functioning generally as described herein.

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Referring to Figs. 9-12, the fluid processing device 10 preferably further includes a liquid channel assembly 150 that includes a generally circular plate 152 disposed generally between the compressor 16 and the pump 18 and a tubular channel member 153 connected with the plate 152 and disposed about the drive shaft 30 so as to extend generally axially therealong. The flow plate 150 has an outer circumferential edge 150a that is disposed against the housing inner surface 25 and is at least partially curved generally upwardly. As such, the liquid flow channel horizontal portion 37 is defined between the flow plate 152 and the compressor lower end wall 124B and has a curved section 37a defined between the plate curved edge 152a and the lower, outer edge of the compressor casing 60. Further, the tubular channel member 153 extends generally downwardly from the plate 152 and provides a generally vertical channel 155 that fluidly couples the fluid passage 34 with the pump inlet 28. Although the flow channel assembly 150 is preferred, the fluid machine 10 may be constructed without such an assembly, such that liquid horizontal channel 37 is defined generally between the compressor casing 60 and the pump casing 80 and liquid portions L flow directly into the pump inlet 28.

Preferably, a baffle or baffle plate 154 is disposed about the drive shaft 30 at the upper end 153a of the tubular channel member 153 so as to be generally centered within the flow plate 152. Further, the baffle 154 has a plurality of openings 156 fluidly coupling the separator 14 and the pump 18, i.e., through the liquid flow passage 34 and vertical channel 155, and is configured to generally distribute liquid portions L flowing to the baffle 154 generally about the pump inlet 28. Furthermore, the fluid processing machine 10 also preferably includes liquid flow regulator assembly 160 configured to vary a flow rate of the liquid portions L from the separator 14 to the pump inlet 28. Preferably, the flow regulator assembly 160 preferably includes an adjustable vane assembly 162 disposed generally between the separator 14 and the pump 18, most preferably mounted to the tubular channel member 153 generally proximal to the member lower end 153b. The vane assembly 162 includes a plurality of rotatable or pivotable vanes 164 mounted to the channel member 153 and an actuator 166, preferably a motor 167, operatively connected with the vanes 164 and configured to rotate the vanes 164. A plurality of openings 165 (Fig. 10) are each defined between each pair of adjacent vanes 164, such that rotation of the blades 164 alternately increases or decreases the size of the openings 165 (and preferably also closes the openings 165), so as to adjust the capacity of the pump 18 dependent on the volumetric flow rate of the liquid portions L to the pump 18. Further, the flow regulator assembly 160 also preferably includes a flow sensor (not shown) coupled

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with the motor 167 and configured to sense the rate of flow through the liquid flow passage 34 and a controller (not shown) coupled with the sensor and the motor 167. The controller is configured to operate the motor 167 to adjust the vane openings 165 to generally directly correspond with the sensed liquid flow rate. In other words, the vanes 5 164 are rotated so as to increase the size of the openings 165 when the liquid flow rate is relatively greater and to alternatively rotate the vanes 164 to decrease or even close the openings 165 when the sensed flow rate is relatively lesser or substantially zero.

Referring to Figs. 9-11, the pump casing 82 preferably includes a generally circular “donut-shaped” main body portion 170 having upper lower surfaces 172A, 172B and an 10 outer circumferential perimeter 173. The pump impeller 84 preferably includes a generally circular hub 176 and a plurality of elongated plates 178 spaced circumferentially about the hub 176 and each providing one of the vanes 83. The plates 178 each extend generally radially and at least partially axially so as to have a generally spiral shape. A plurality of flow passages 180 are defined between each pair of adjacent vane plates 178 15 and each has an inlet 180a fluidly connected with the casing suction nozzle 87 and an outlet 180b at the hub outer circumferential perimeter 176a. As such, liquid portions L flowing into the suction nozzle 87 enters into one or more of the passage inlets 180a are accelerated by the rotating vanes 83 to flow both radially and tangentially outwardly from the passage outlets 180b so as to “swirl” in a generally spiral manner into the volute 20 chamber 89. The liquid portions L decelerate within the volute chamber 89 such that the liquid pressure is increased, and then the pressurized liquid portions L_P flow out of the discharge nozzle 88 into the housing liquid outlet 24B, thereafter merging with the pressurized gas portions G_P flowing out of the housing gas outlet 24A, as discussed above.

Although a centrifugal pump 80 is presently preferred, the pump 18 may 25 alternatively be any other appropriate type of rotary pump, such as a gear pump or a screw pump, a reciprocating pump such as a piston pump, or any other type of pump (none shown).

It will be appreciated by those skilled in the art that changes could be made to the embodiments described above without departing from the broad inventive concept thereof. 30 It is understood, therefore, that this invention is not limited to the particular embodiments disclosed, but it is intended to cover modifications within the spirit and scope of the present invention as defined generally in the appended claims.

I claim:

1. A fluid processing machine for processing a multiphase fluid stream including a mixture of at least a gas and a liquid, the fluid processing machine comprising:
 - 5 a housing having an interior chamber, an inlet fluidly connected with the interior chamber and fluidly connectable with a source of the multiphase stream, and first and second outlets; a separator disposed within the housing chamber, fluidly coupled with the inlet such that the fluid stream flows generally to the separator, and configured to generally separate the fluid stream into a substantially gaseous portion and a substantially
10 liquid portion;
 - a compressor disposed within the housing chamber and being fluidly coupled with the separator such that the gaseous portion flows into the compressor, the compressor being configured to compress the gaseous portion and to discharge the compressed gaseous portion through the housing first outlet; and
15 a pump disposed within the housing chamber and having an inlet fluidly coupled with the separator, the pump being spaced generally vertically from the separator such that the liquid portions flow at least partially by gravity from the separator to the pump inlet, the pump being configured to pressurize the liquid portion and to discharge the pressurized liquid portion through the housing second outlet.
20
2. The fluid processing machine as recited in claim 1 wherein the compressor is one of:
 - disposed generally between the separator and the pump; and
25 disposed generally above the separator such that the separator is disposed generally between the compressor and the pump.
3. The fluid processing machine as recited in claim 1 wherein the housing includes an inner surface at least partially defining the housing chamber and the compressor has an outer surface spaced from the housing inner surface such that at least one liquid flow
30 passage is at least partially defined between the two surfaces, the liquid flow passage fluidly coupling the separator with the pump.

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4. The fluid machine as recited in claim 3 wherein the housing has a centerline, the housing inner surface extends circumferentially about the centerline, the compressor has a central axis, and the compressor outer surface extends circumferentially about the compressor central axis and is spaced radially inwardly from the casing outer surface such
5 that at least a portion of the liquid flow passage is generally annular.

5. The fluid processing machine as recited in claim 4 wherein the flow passage includes a generally horizontal, generally radially extending portion defined between the compressor and the pump and fluidly coupled with the pump inlet.
10

6. The fluid processing machine as recited in claim 3 wherein at least a portion of the liquid flow passage extends generally vertically such that gravity at least partially initiates flow of the liquid through the flow passage.

7. The fluid processing machine as recited in claim 3 the separator is configured to direct liquid generally toward the housing inner surface such that liquid flows into the liquid flow channel and thereafter flows at least partially by gravity to the pump inlet.
15

8. The fluid processing machine as recited in claim 1 further comprising a shaft extending generally vertically through the housing chamber and being rotatable about a central axis, each one of the separator, the compressor, and the pump having at least one rotatable member connected with the shaft such that shaft rotation generally operates each one of the separator, the compressor and the pump, the rotatable members being spaced apart vertically along the shaft axis.
20

9. The fluid processing machine as recited in claim 8 further comprising a motor connected with the shaft and configured to rotate the shaft about the central axis.
25

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10. The fluid processing machine as recited in claim 1 wherein the separator includes a body rotatable about a central axis, the separator body having a first end with an opening fluidly coupled with the housing inlet so as to receive the fluid stream, a second end with an opening fluidly coupled with the compressor, and an inner separation surface extending
5 circumferentially about the axis and generally between the body first and second ends, the separation surface defining a separation chamber and being angled generally radially outwardly in a direction toward the body first end such that as the separator body rotates about the axis, liquid portions of the fluid stream contacting the separation surface are directed generally radially outwardly away from the body axis and generally axially
10 toward the body first end so as to flow through the body first opening while a remainder of the fluid stream flows through the body second opening.

11. The fluid processing machine as recited in claim 10 wherein:
the housing has a generally vertically extending inner surface at least partially
15 defining the interior chamber;
the separator body first end is spaced generally vertically upwardly with respect to the housing inlet; and
the separator body further has an outer separation surface extending
circumferentially about the axis and generally between the body first and second ends, the
20 outer separation surface being angled generally radially outwardly in a direction toward the body first end such that as the separator body rotates about the axis, liquid portions of the fluid stream contacting the outer separation surface are directed generally radially outwardly away from the body axis and generally axially toward the body first end so as to be directed generally toward the housing inner surface.

25

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12. The fluid processing machine as recited in claim 10 wherein the separator body includes:

an inner tubular hub member;

an outer tubular member connected with and disposed generally coaxially about the hub, the tubular member having an upper, generally circular inlet end with a first diameter, a lower, generally circular outlet end with a second diameter, the first diameter being larger than the second diameter, and a generally conical inner surface providing the separation surface, the conical surface extending circumferentially about the hub and facing generally toward the upper end; and

a plurality of elongated vanes extending between and connecting the inner and outer tubular members.

13. The fluid processing machine as recited in claim 1 wherein the compressor has a casing with an interior chamber and an inlet opening extending into the chamber and the separator is disposed within the casing opening such that gas portion flows from the separator directly into the compressor chamber.

14. The fluid processing machine as recited in claim 13 wherein:

the separator includes a body rotatable about a central axis, the separator body having a first end with an opening fluidly coupled with the housing inlet so as to receive the fluid stream and a second, opposing end with a gas outlet opening; and

the compressor has at least one impeller disposed within the casing chamber and having an inlet, the separator body second end being disposed generally adjacent to the impeller such the gas portion flows from the separator gas outlet to the impeller inlet.

15. The fluid processing machine as recited in claim 1 wherein the housing has a central axis and includes a generally concave wall disposed within the interior chamber so as to define a plenum chamber, the separator being rotatable about the central axis and disposed at least partially within the plenum chamber, the housing plenum wall being arranged with respect to the inlet such that multiphase fluid stream is directed to flow at least one of radially about the axis and generally parallel with respect to the axis prior to entering the separator inlet.

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16. The fluid processing machine as recited in claim 1 wherein the compressor includes:
- a casing disposed within the housing and defining an interior chamber, the casing having an inlet fluidly coupled with the separator and an outlet fluidly coupled with the housing first outlet; and
 - at least one rotatable impeller disposed within the casing chamber and configured to compress gaseous fluid portions flowing into the compressor casing inlet.
17. The fluid processing machine as recited in claim 16 wherein the compressor has a plurality of compression stages, each compression stage includes one rotatable impeller and one fixed flow channel assembly disposed about the impeller, the flow channel assembly including a diffuser channel, a return bend channel, a return channel and a guide vane.
18. The fluid processing machine as recited in claim 1 wherein:
- the compressor casing includes a generally tubular wall having an outer circumferential surface; and
 - the housing has an inner circumferential surface, the casing inner surface being spaced radially inwardly from the housing inner surface so as to form a generally annular liquid flow passage, and a plurality of elongated struts extending between the housing inner surface and the casing outer surface and configured to connect the casing with the housing.
19. The fluid processing machine as recited in claim 1 wherein the pump includes:
- a casing disposed within the housing and having an interior chamber, an inlet opening providing the pump inlet, and an outlet opening fluidly coupled with the housing second outlet; and
 - a rotatable impeller disposed within the casing chamber and configured to compress liquid flowing into the pump inlet and to direct the fluid toward the casing outlet opening.

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20. The fluid processing machine as recited in claim 19 further comprising a baffle disposed generally between the compressor and the pump and having a plurality of openings fluidly coupling the separator and the pump, the baffle being configured to generally distribute liquid flowing to the baffle generally about the pump inlet.
- 5
21. The fluid processing machine as recited in claim 20 further comprising an adjustable vane assembly disposed generally between the separator and the pump and configured to vary a flow rate of liquid portions from the separator to the pump inlet.
- 10
22. A fluid processing machine for processing a multiphase fluid stream including at least gaseous and liquid fluid portions, the fluid processing machine comprising:
- a housing including an interior chamber, a central axis extending through the chamber, an inner surface extending circumferentially about the axis and at least partially defining the chamber, an inlet fluidly connected with the housing chamber and configured
- 15 to receive the multiphase stream, and first and second outlets fluidly connected with the chamber;
- a separator disposed within the housing chamber, fluidly coupled with the inlet such that the fluid stream flows generally from the inlet to the separator, and configured to generally separate the fluid stream into a substantially gaseous portion and a substantially
- 20 liquid portion;
- a compressor disposed within the housing chamber so as to be spaced axially from the separator, being fluidly coupled with the separator so as to receive the gaseous portion from the separator and having an outlet fluidly connected with the housing first outlet, the compressor being configured to compress the gaseous portion and to discharge the
- 25 compressed gaseous portion through the housing first outlet, the compressor having an outer surface spaced inwardly from the housing inner surface so as to define at least one flow passage extending generally parallel with the housing axis; and
- a pump disposed within the housing chamber so as to be spaced axially from the separator such that the compressor is disposed generally between the separator and the
- 30 pump, the pump having an inlet fluidly coupled with the separator through the at least one flow passage so as to receive the liquid portion from the separator, the pump being configured to pressurize the liquid portion and to discharge the pressurized liquid portion through the housing second outlet.

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23. The fluid processing machine as recited in claim 22 wherein the compressor has a central axis extending generally collinearly with the housing axis and the casing outer compressor outer surface extends generally circumferentially about the compressor axis such that the flow passage is generally annular.

5

24. The fluid processing machine as recited in claim 23 further comprising a plurality of a plurality of support members spaced circumferentially about the housing axis and configured to mount the compressor within the housing, each adjacent pair of support members defining a separate generally arcuate flow passage section of the flow passage.

10

25. The fluid processing machine as recited in claim 22 wherein the housing has a generally circular tubular body providing the housing inner surface and the compressor includes a generally circular cylindrical casing disposed within the housing body and providing the casing inner surface such that the flow passage is generally annular.

15

26. A fluid processing machine for processing a multiphase fluid stream including at least a gaseous portion and a liquid portion, the fluid processing machine comprising:

20 a housing including an interior chamber, an inner circumferential surface at least partially defining the chamber, an inlet extending into the chamber and configured to receive the multiphase stream, and two outlets extending out of the chamber;

a separator disposed within the housing chamber, fluidly coupled with the inlet such that the fluid stream flows to the separator, and configured to generally the fluid stream into substantially gaseous portion and a substantially liquid portion;

25 a compressor disposed within the housing chamber generally beneath the separator, having an inlet fluidly coupled with the separator so as to receive the gaseous flow from the separator, and configured to compress the gaseous flow and to discharge the compressed gaseous flow from the housing through the first outlet, the compressor further having an outer surface spaced inwardly from the housing inner surface such that a generally annular flow channel is defined between the two surfaces, the flow channel
30 being fluidly coupled with the separator; and

a collector chamber section defined within the housing chamber generally beneath the separator, the collector chamber being fluidly coupled with the annular flow channel and with the housing second outlet, the collector chamber being configured to contain a quantity of liquid.

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27. A fluid processing machine for processing a multiphase fluid stream including a mixture of at least a gas and a liquid, the fluid processing machine comprising:

a housing having an interior chamber, an inlet fluidly connected with the interior chamber and fluidly connectable with a source of the multiphase stream, and first and second outlets; a separator disposed within the housing chamber, fluidly coupled with the inlet such that the fluid stream flows generally to the separator, and configured to generally separate the fluid stream into a substantially gaseous portion and a substantially liquid portion;

a compressor disposed within the housing chamber, having a casing with an interior chamber and an inlet extending into the casing chamber, and at least one impeller disposed within the casing chamber and configured to compress the gaseous portion, the compressor being configured to discharge the compressed gaseous portion through the housing first outlet, at least a portion of the separator being disposed within the casing inlet such that the gas portion flows generally directly from the separator into the compressor chamber; and

a pump disposed within the housing chamber, having an inlet fluidly coupled with the separator so as to receive the liquid portion, and configured to pressurize the liquid portion and to discharge the pressurized liquid portion through the housing second outlet.

28. The fluid processing machine as recited in claim 27 wherein the pump is spaced generally vertically from the separator such that liquid portions flow at least partially by gravity from the separator to the pump inlet.

29. The fluid processing machine as recited in claim 28 wherein the housing has an inner surface at least partially defining the housing chamber, the compressor has an outer surface spaced inwardly from the housing inner surface so as to define at least one flow passage extending generally parallel with the housing axis, and the pump inlet is fluidly coupled with the separator through the at least one flow passage so as to receive the liquid portion from the separator.

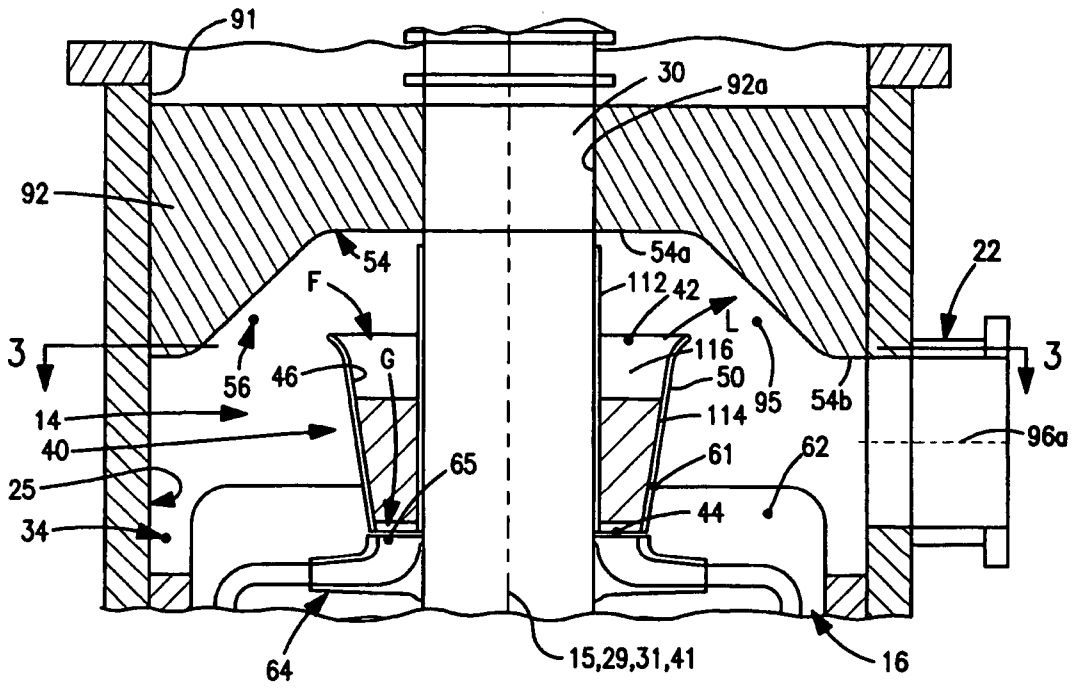


FIG. 2

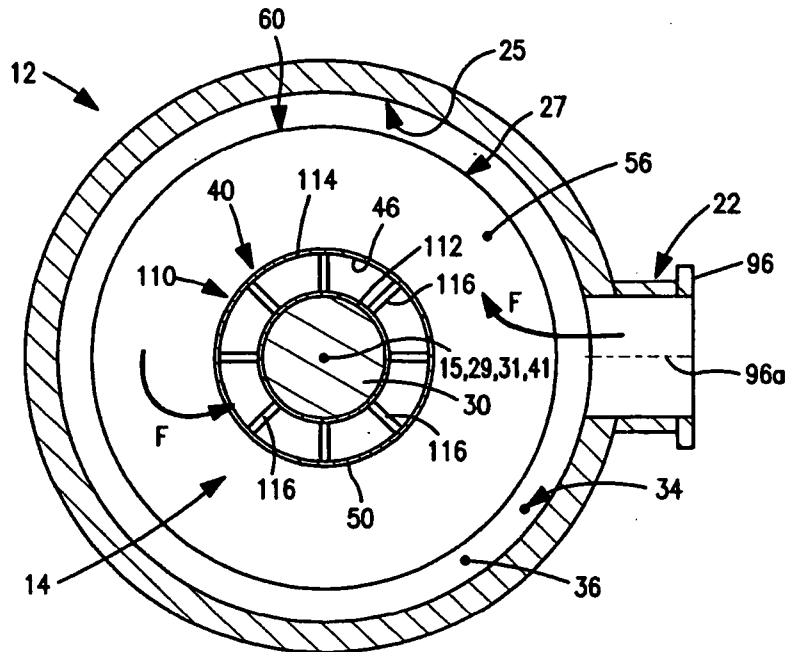


FIG. 3

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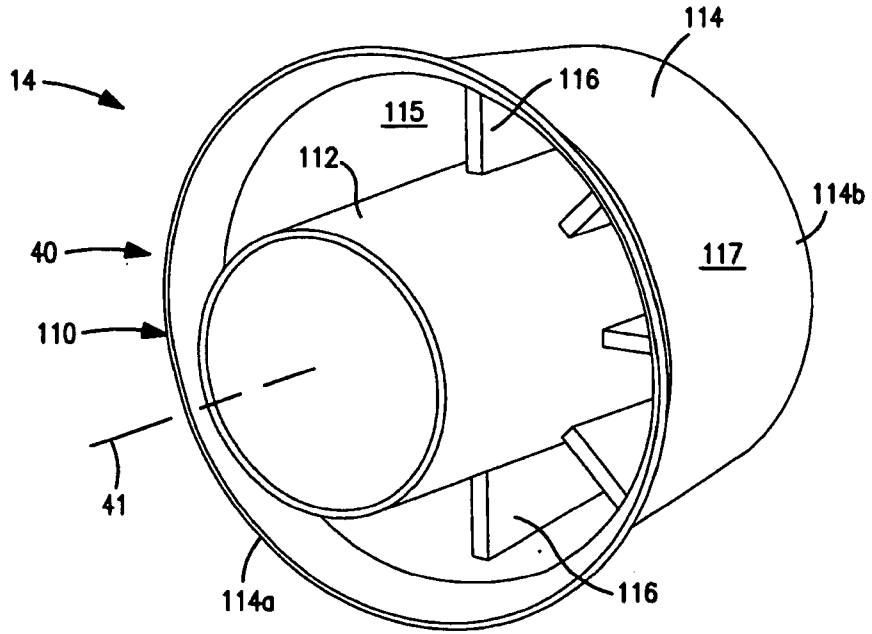


FIG. 4

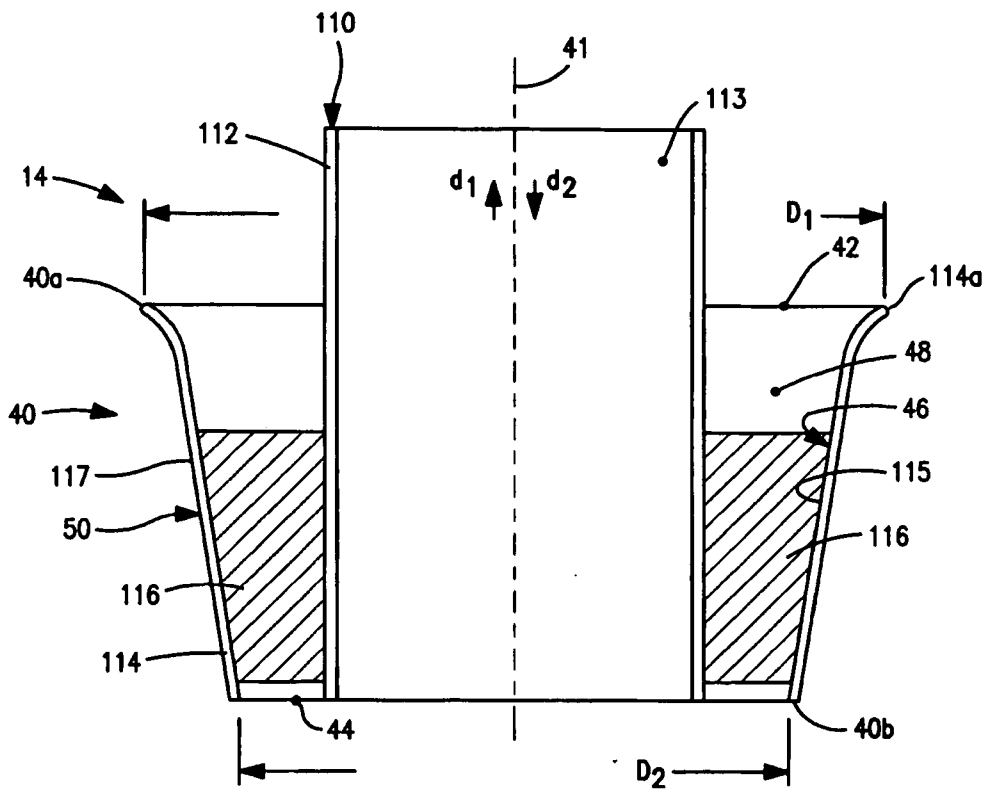


FIG. 5

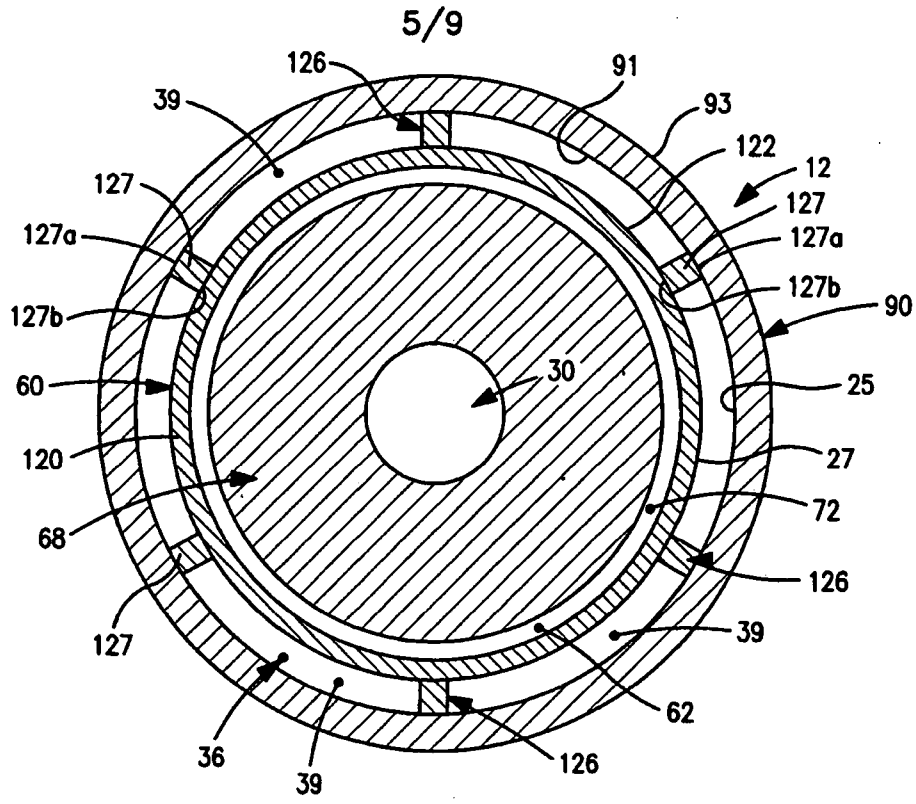


FIG. 7

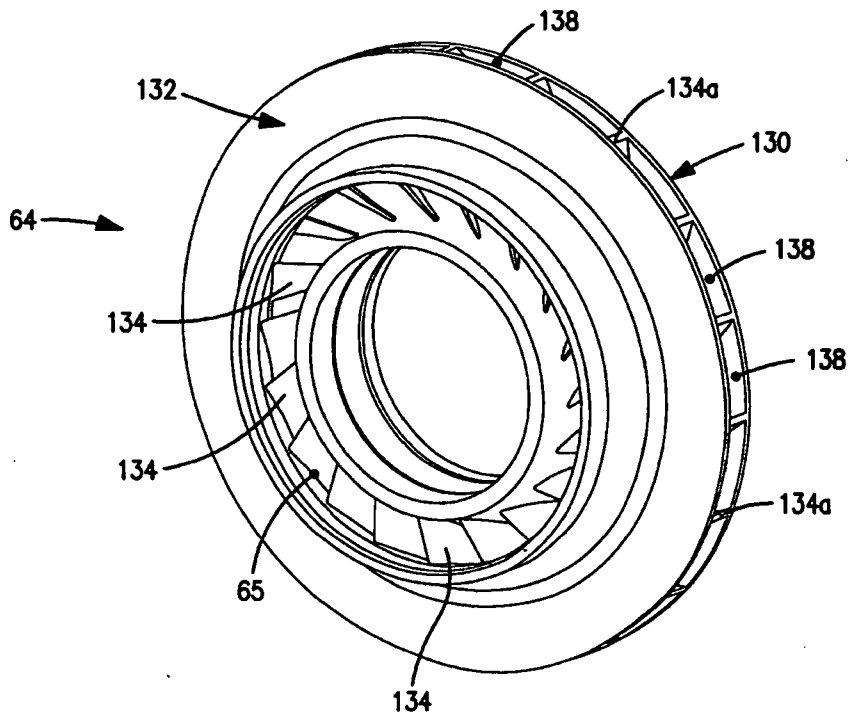


FIG. 8

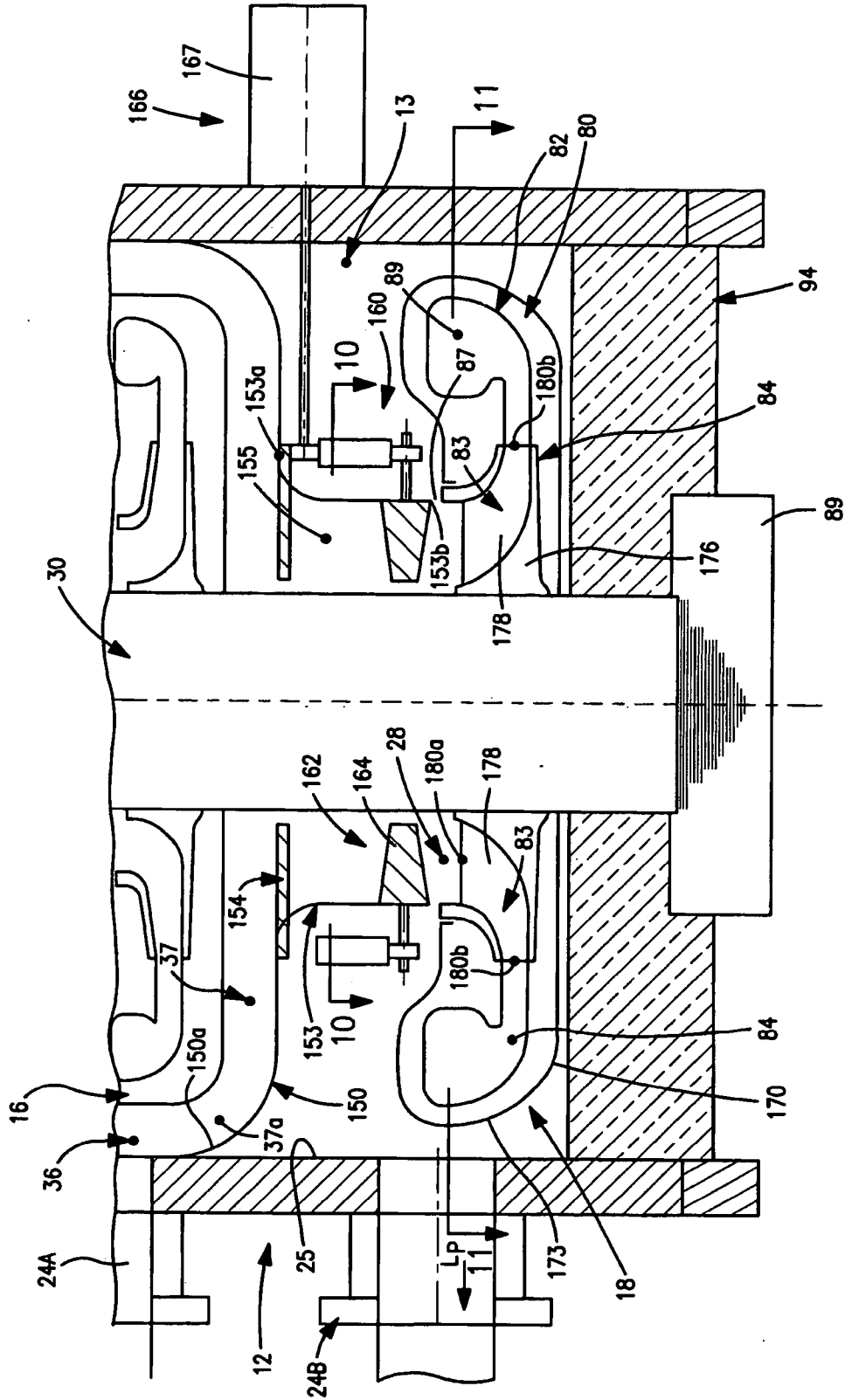


FIG. 9

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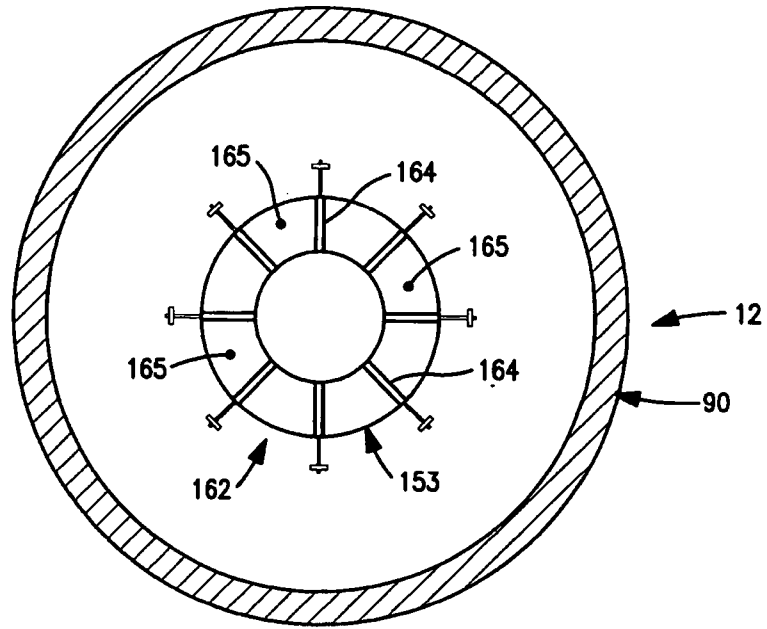


FIG. 10

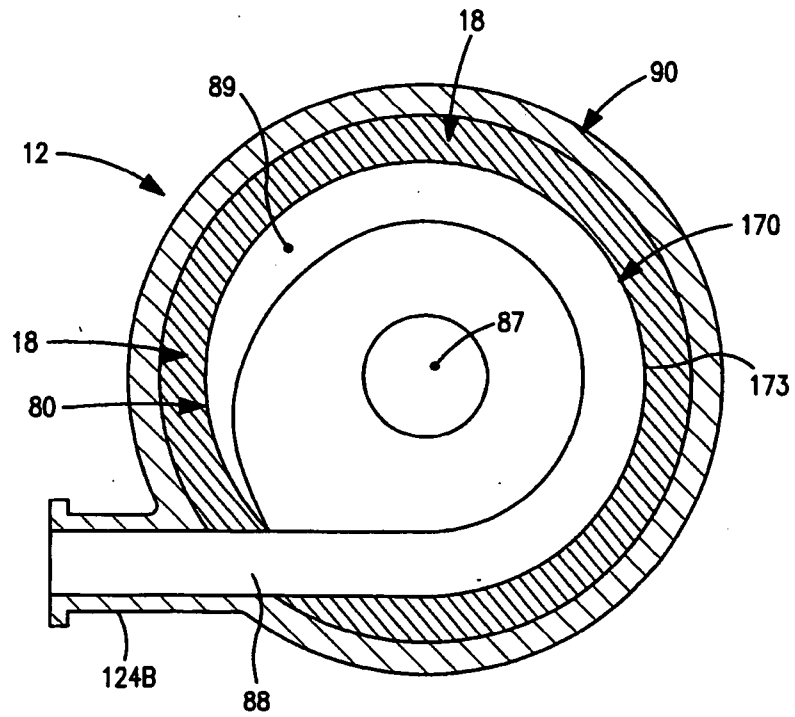


FIG. 11

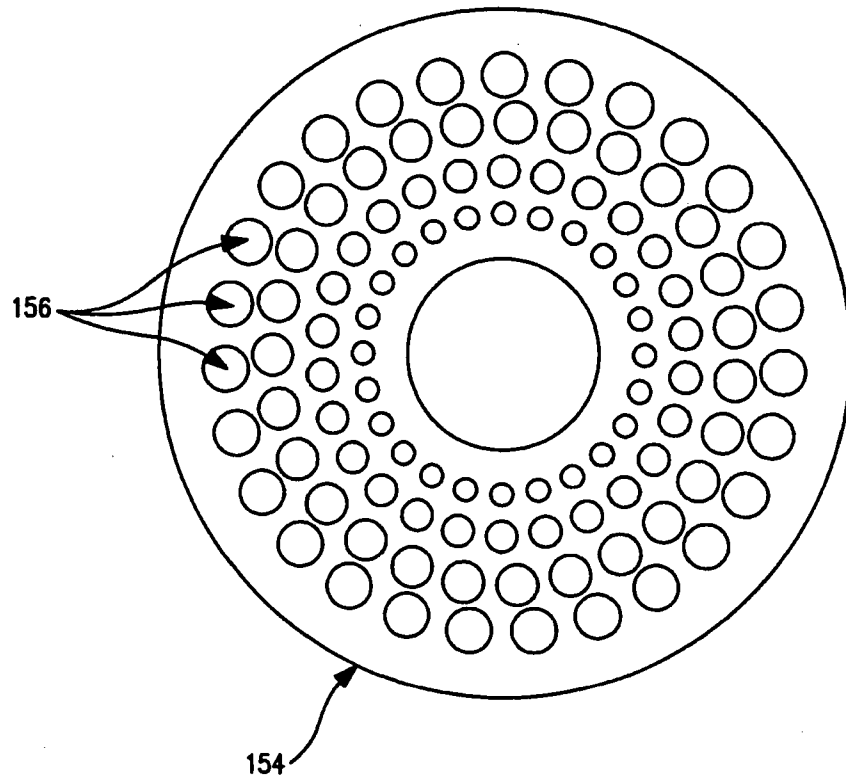


FIG. 12

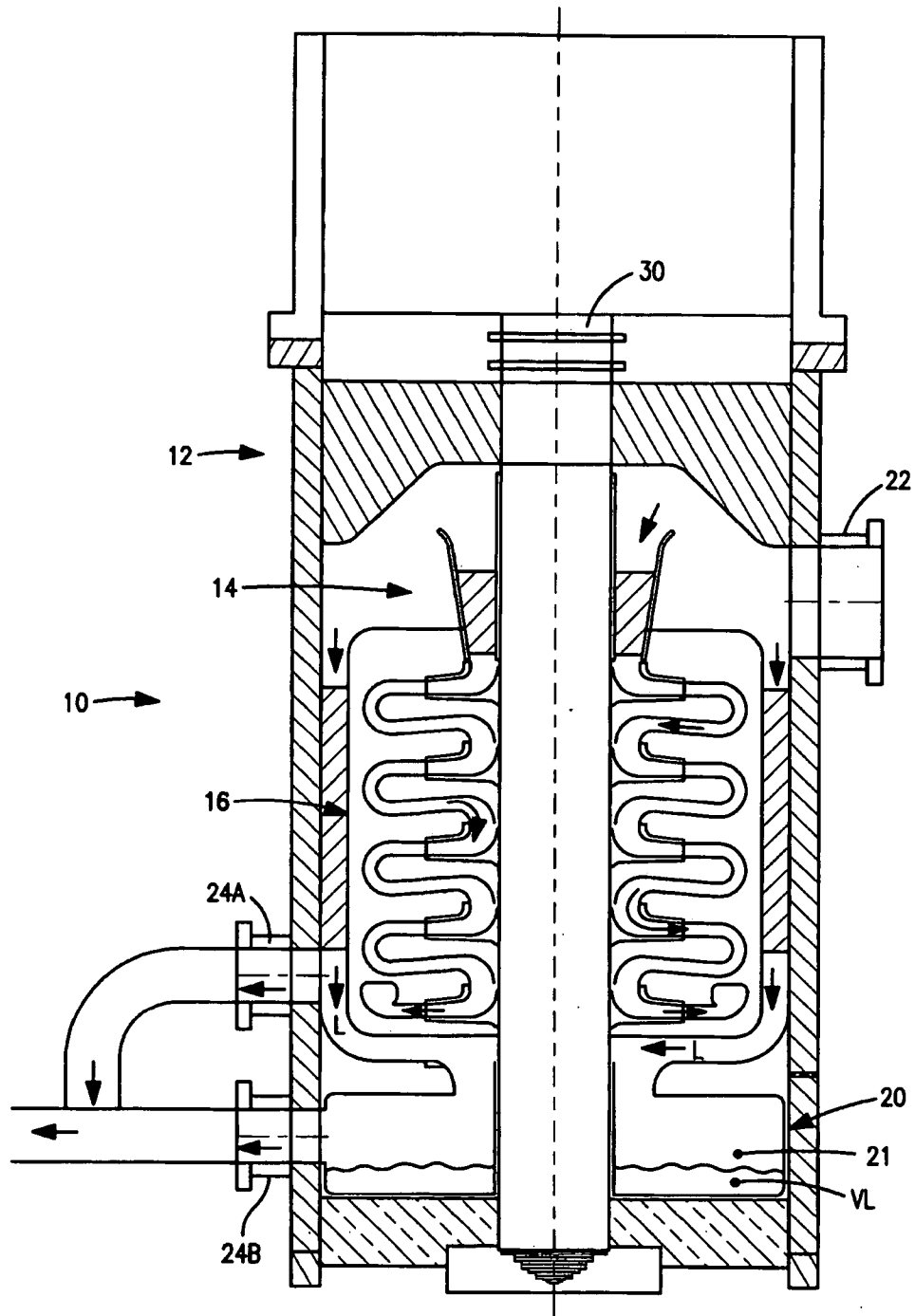


FIG. 13