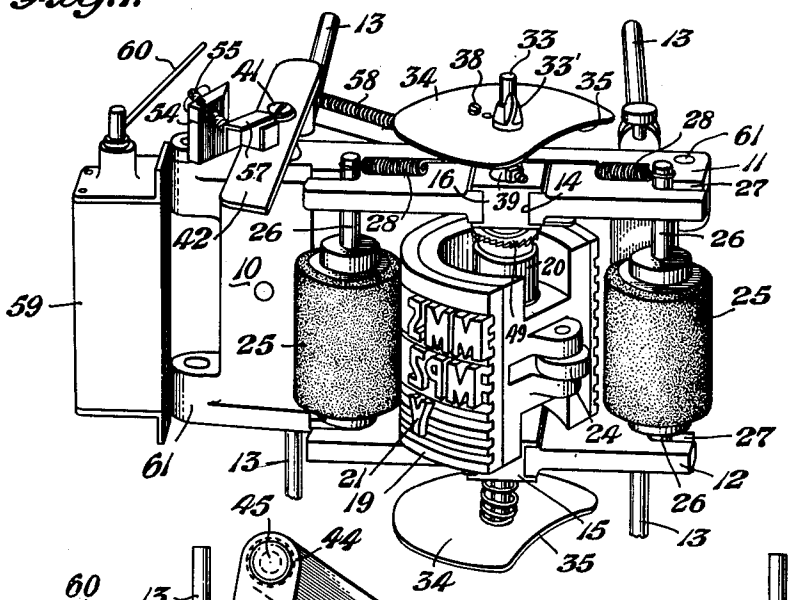
April 28, 1953 M. MANN 2,636,436

CONVEYER MARKER

Filed June 8, 1949 2 SHEETS—SHEET 1

Fig. 1.



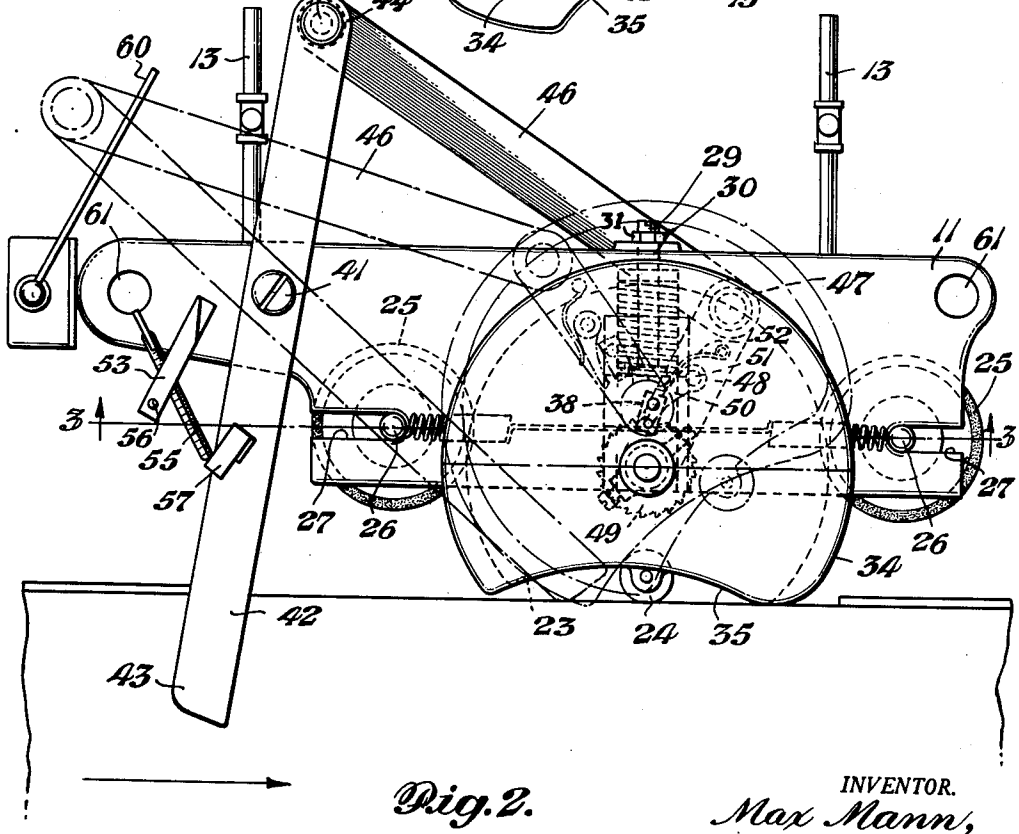


Fig. 2.

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CONVEYER MARKER

Filed June 8, 1949

2 SHEETS—SHEET 2

Fig. 3.

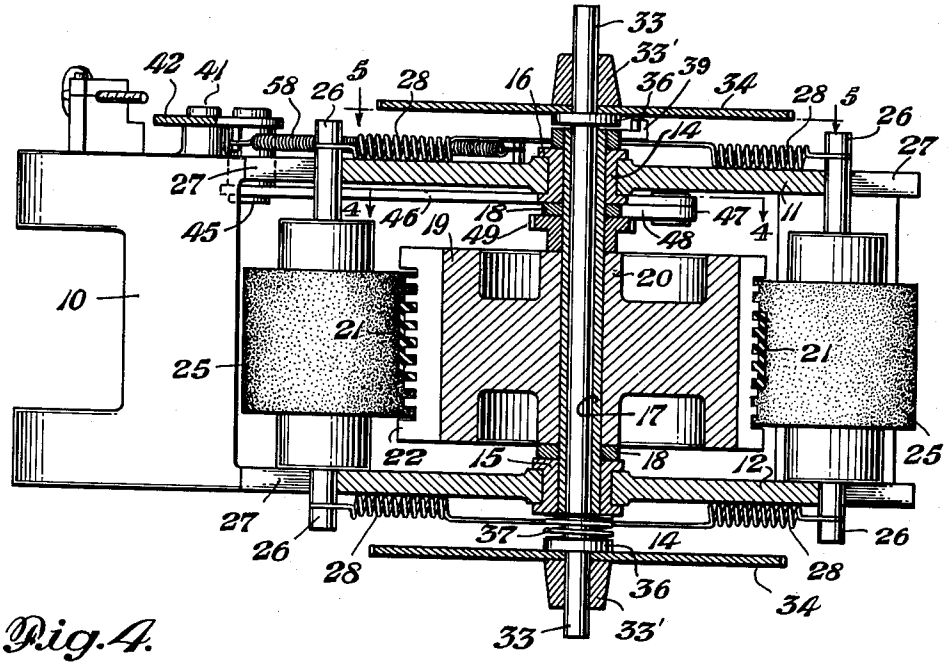


Fig. 4.

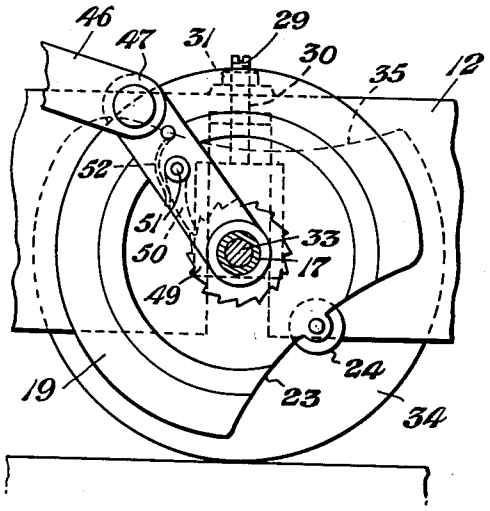
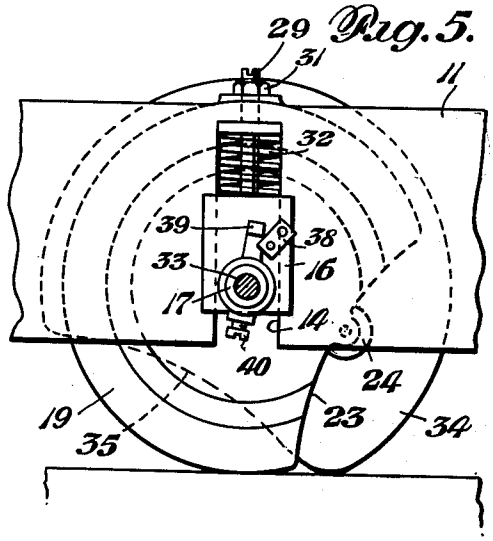


Fig. 5.



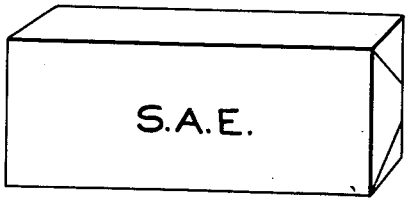


Fig. 6.

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UNITED STATES PATENT OFFICE

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CONVEYER MARKER

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Application June 8, 1949, Serial No. 97,763

17 Claims. (Cl. 101—235)

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The present invention relates to a marking device comprising a printing stamp having a rotary die with controlled spacing elements actuated by a package, to print upon the package in a manner to impart the print in a definite position upon the package as it moves by the marker on a conveyor to give only a single imprint at an adjustable and predetermined position thereon.

Various markers of the art have generally comprised rotary printing dies which continuously rotate in contact with the article moving by, thereby giving a continuous print across the surface as it moves by, and moreover, many times such print when applied in the same plane as the moving surface to be printed upon is imperfect, smudged, or incomplete.

In contrast, the present marker is resilient and actuated by the moving package itself so that it will only print when the package is in position before the marker to receive the print and has a spacing element to control the printing die. Since the moving package actuates both a spacing element and the printing die the packages may be spaced closely or at wide intervals on the conveyor and the print is imparted upon the package in a predetermined space which may be varied by adjustment of the marking device so that each package will receive the print in the particular pre-selected spot.

It is accordingly an object of the present invention to provide actuation means for a printing die by a moving article to be marked, the marking being effected by friction of the moving body against the marker.

It is a further object of the present invention to provide spacing means for gripping the moving package providing delayed printing action and adjustable to measure the exact point at which printing action will begin.

It is a further object to provide a resiliently mounted rotating marking element to conform to the plane of the surface of the body to be marked and impart thereto, by rolling action, a clearly legible print.

It is a further object to provide a resiliently mounted rotary printing die constructed to stop after a single print.

Other objects are inherent in the construction shown and of which various modifications will occur to one skilled in the art; the present construction shown being illustrative for purposes of explaining the principles of the present invention. The invention will be best understood by reference to the drawings herein of which,

Fig. 1 shows a perspective view of the assembled marking device which will be mounted beside a conveyor.

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Fig. 2 is a plan view of the device showing operation of both the spacer and printing die in alternate positions.

Fig. 3 is a sectional elevation of the marker taken on the lines 3—3 of Fig. 2.

Fig. 4 is a detailed drawing showing the spacer element in contact with the package to be printed while the printing die is in inactive position but cocked and ready for printing, taken on the line of 4—4 Fig. 3.

Fig. 5 is an analogous detail in sequence following Fig. 4 showing the printing die beginning to print, the spacer element being rotated to inactive position, taken on the line 5—5 of Fig. 3.

Fig. 6 shows a package with a single centered printing thereon, as would be formed by use of the present marker.

The device comprises a framework 10, having upper and lower plate-like supporting element 11 and 12 generally cast as an integral part of the frame 10, suitably supported on adjustable rods 13, to be held rigidly beside a conveyor. The upper and lower plates 11 and 12 are slotted and machined at 14 with horizontal slots opening from the forward printing face of said plates and extending through most of the surface of the plate, to receive in a sliding fit the vertically supported printing and spacing assembly between sliding supports 15 and 16 through an elongated sleeve 17 rotatably mounted in the supports 15 and 16 and journaled in bushings 18 to rotatably support said sleeve 17, the entire assembly being thereby radially slidable in said sliding groove 14 while being rotatable about sleeve 17.

Centrally mounted about and rigidly fixed to sleeve 17, and supported through a hub 20, is a cylindrical printing element 19 carrying on the periphery thereof printing indicia of rubber or other suitable printing die element materials 21. The peripheral outer edge 22, thereof may likewise be of rubber or may be of serrated metal adapted to form a guide capable of gripping and frictionally rotating the cylindrical printing die 19 with the moving surface to be printed upon. The cylindrical printing element 19 has a curved segment 23, cut therefrom to form a hollowed outer portion of discontinuity of the cylindrical printing surface, whereby on frictional rolling contact of the printing die with the surface to be printed rotation will be discontinued when the hollowed out portion is reached, as shown in the dotted line section of Fig. 2, at which point the

package to be printed engages a roller 24, suitably journaled in the cutaway face of cylinder 19 to bypass the cylindrical printing element 19 without causing further rotation thereof, the roller 24 at this position bearing against the package surface allowing it thus to roll by after printing.

Slidable supports 15 and 16 are adjustably fixed by pins 29, mounted axially of the sliding slots 14 bored and threaded into the inner ends of sliding supports 15 and 16, said pins 29 further being slidable through holes 30, bored axially of the slots 14 in the plates 11 and 12. The threaded pins 29 each carry an outer stop collar 31 adjustably threaded thereon which defines the limit at which supports 15 and 16 may extend outwardly from the slots 14. Supports 15 and 16 are biased outwardly of said slots 14 against stop 31 by compression springs 32, mounted in slots 14, and about pins 29, bearing against the supports 15 and 16 and the inner end of slots 14 to resiliently hold the supports in extended position outwardly of said slots.

Mounted centrally within sleeve 17, is a rotary shaft 33 and supporting at each end spacer discs 34 of circular contour, each disc having a curved segment 35 cut from the periphery thereof to define an inactivated position of the discs in contact with the package surface, in which position the discs do not rotate as shown in Fig. 2. The peripheries of the discs 34 are serrated to allow frictional engagement by the package surface and rotation thereof and the shaft 33 to which they are pinned to be rotated thereby as the package moves by. The shaft 33 and discs 34 are resiliently supported in sleeve 17 on washers 36 and spring 37 whereby there is substantially free and independent rotation of both discs as a unit with respect to the rest of the printing assembly.

Each of the spacer discs is radially aligned with each other so that the cutaway portions 35 lie in the same vertical plane, and are rigidly fastened to shaft 33 in this position by any suitable fastening elements 33' to rotate together.

One of the discs 34 carries an abutment 38 rigidly fixed thereto on the under side thereof. Pinned to one end of sleeve 17 for rotation therewith is an adjustable dog or protuberance 39 which may be fixed in any suitable position about said sleeve 17 through set screw 40. Thus in adjustable position, the dog 39 protrudes out and extends into the radial path of shoulder 38 on disc 34 so that after sufficient rotation of sleeve 17 the dog 39 will engage the shoulder 38 so that when the printing element and sleeve 17 on which it is mounted is rotated in one direction, the dog 39 ultimately engages shoulder 38 of the friction discs 34 and rotates the same therewith. Continuous rotation of the friction discs 34 alone will allow shoulder 38 to disengage from said dog 39 and the friction discs spacer assembly may, therefore, rotate almost 360 degrees before again engaging the dog 39 and rotating the printing element attached to sleeve 17 therewith.

Mounted on each side of the cylindrical printing element 19 are a pair of inking rollers 25 having a surface of conventional ink absorbent pad material rotatably supported through shafts 26 in aligned grooves 27 in both upper and lower plates 11 and 12 for rotary support and easy demountability. The ends of each of the shafts 26 are engaged by tension springs 28 resiliently urging both linking rollers 25 into inking contact with the printing die surface 21.

Pivotally mounted above plate 11 through pivot 41 is a trip or actuating arm 42 which is of sufficient length to extend into the path of the conveyor and be engaged by a package passing by thereon, said trip arm 42 being moved thereby on its pivot 41 to fold beneath the upper spacing disc 34 as the surface of the package engages the same, said trip arm having its outer corner 43 rounded to allow the package to pass smoothly thereby by sliding movement thereof. The inner end 44 of said trip arm has pivoted thereto through a pivot 45 substantially elongated as is shown in Fig. 3 to pivotally carry therewith link arm 46 which in turn is pivoted at its opposite end 47 to a sub link 48, the inner end of which is integral with bushing 18 mounted for easy rotation about sleeve 17. Sleeve 17 has mounted about and rigidly fastened thereto, for rotation therewith ratchet gear 49 mounted directly beneath bushing 18. Link arm 48 carries beneath its surface a pawl 50 pivotally supported thereto at 51, said pivoted construction supporting pawl 50 in direct engagement with ratchet gear 49, said pawl being resiliently constrained in continuous engagement with said gear 49 by hairpin spring 52.

With the lever and link construction, as thus described, actuation of trip arm 42 by the package, causes it to swing on pivot 41 drawing links 46 and 48 as shown in the dotted line portion of Fig. 2. Such actuation by a moving package against said trip arm 42 and its links 46 and 48 interconnected through ratchet 49 and pawl 50 will cause sleeve 17 and the rotary printing cylinder 19, fastened thereto, to rotate through a short arc adjustable as hereinafter explained. As dog 39 upon sleeve 17 may be in direct bearing contact with shoulder 38, rotation of said sleeve 17 through actuation of this link and lever system, will also cause spacer discs 34 to rotate a similar distance or a lesser distance, depending upon how far dog 39 must be rotated before it begins to bear against stop 38. It can thus be seen that by proper setting of the dog 39 on sleeve 17 a certain amount of lost motion may be obtained upon rotation of the printing cylinder 19 through the link and lever system prior to actual activation of the spacer element, i. e. the number of degrees the dog 39 on sleeve 17 must be rotated prior to bearing on stop 38 in spacer disc 34 for rotation of the same.

A shoulder 53 is provided outside of the arc of movement of lever 42 on plate 11 and is bored and threaded in its upper portion 53 and split from the side 54 into the bore for receiving an adjustable stop screw 55 and clamping by set screw 56 the same therein in adjustably fixed position. Cooperating with said adjustable stop screw 55 is an abutment 57 rigidly mounted on said lever 42. A spring 58 is mounted in tension between the inner end of lever 42 near pivot 44 and plate 11 and resiliently constrains lever 42 and the link system pivoted thereto to its extended trip arm position and against stop set screw 55 through shoulder 57 to constrain the link and lever system described, to open operative position ready for activation by a moving package as shown in the full line position of Fig. 2. It will be noted from this construction that movement of stop screw 55 allows adjustment of the portion of trip lever 42 to short or long arcuate movement and has the practical effect of controlling rotation of the printing die 19 and spacer discs 34 ultimately movable therewith through dog 39 and stop 38.

As thus described spacer discs 34 are of larger

diameter than the printing cylinder 19 except at the position of the cutaway segment 35. The whole assembly of both spacer discs 34 and printing cylinder 19 is resiliently slidable in groove 14.

In actual operation, therefore, any package presenting a flat printable surface such as a box or carton passing along the conveyor will first engage the trip arm 42 and depress the same through an arc swinging at its pivot 41 as shown in dotted line portions of Fig. 2. As trip arm 42 is depressed through the links 46 and 48 attached thereto, and with pawl 50 engaging ratchet gear 49, the entire assembly, printing cylinder 19 and spacer discs 34, will be rotated through a short arc sufficient to present the frictionally rotatable peripheral edges to the package for rotation as adjusted both by the stop screw 55 maintaining the initial position of lever 42 and the set relationship of the dog 39 which may bear directly against shoulder 38 or may have some pre-adjusted lost motion prior to bearing thereon depending upon the spacing desired. After full tripping of trip lever 42, the package has passed into this position as shown both in the dotted line portion of Fig. 2 and the partially rotated portion of Fig. 1.

The spacer discs 34 being of greater diameter and therefore protruding farther from the assembly than the printing cylinder 19 will be first engaged by the package and the instrument will be so set as to project the spacing discs into the path of the package to be printed. Since the whole assembly is resiliently mounted in sliding groove 14 the exact position of the package is relatively unimportant and the package engaging the assembly will resiliently grip both spacing discs at the surface to be printed and resiliently back the entire assembly along the sliding grooves 14 to a point where both spacing discs 34 through their serrated edges firmly engage the surface to be printed on the package and will begin to rotate as the package moves by. Shoulder 38 upon disc 34 will leave dog 39 after spacer discs begin to rotate on the surface of the package, and, since the printing die 19 is of lesser diameter, the package will not be touched thereby while spacer discs rotate on the moving package, but will remain in the cocked position ready for printing in the position set by the original tripping of the lever 42 in the manner shown both in Figs. 1 and detailed Fig. 4.

When the spacer elements 34 have rotated throughout their normal arc by frictional passage of the package surface, and when the spacer disc diameter begins to recede as the cutaway segment 35 is reached, rotation of spacers will cease. At this point of decreased diameter the spacer merely slides on the moving package surface and the whole rotating assembly, actuated by springs 32 will move outward against the package and the printing cylinder 19 will first engage the surface of the package for printing the substance corresponding to the surface of the printing die 21. In this position the serrated edge or gripping edges 22 frictionally grip the package and the entire printing die 19 now rotates with the moving surface, spacer discs 34 remaining inactivated, as shown in Fig. 5. As the package moves by, therefore, printing die 19 will be rotated until the cutaway portion 23 thereof, comes into contact with the surface of the package, at which point the surface of the package will bear against roller 24 and the entire printing assembly will then remain inactivated; the package rolling by, bearing only against the roller 24, with a minimum of friction.

It will be appreciated by use of spacer elements of various diameters any desired spacing is possible. By adjustment of dog 39 and by adjustment of stop screw 55, great flexibility in positioning of the print is possible with a given size of spacer while applying a single imprint upon each package. The exact spacing of packages upon the conveyor is unimportant so long as they are separated by some space sufficient to release trip arm 42 for recocking by the next package, which will be printed upon, whatever the rate of travel past the marker, substantially at the center thereof or in a pre-adjusted position before or after the center, as desired. Thus, a single imprint is imparted by the device as shown to each package and since the printing element is resiliently mounted, a clear print is invariably obtained. The marker, itself may be supported at various heights by adjustment of rods 13 beside the conveyor (not shown) and if desired a counter element 59 may be mounted to the frame 10 with a projecting arm 60 placed in the path of lever 42 to trip the same and thus derive a count of the number of packages printed, if desired.

The printing die is linked by both rollers 25 as it rotates. Various other means of supporting the framework 10 may be used, for example, it may be supported through bored ears 61 at a suitable position beside the moving conveyor. The construction herein shown is practical and the entire assembly is easily demounted for cleaning or replacing of printing elements or spacing discs by release of set pins 29. Both inking rollers 25 are easily removed through slots 27 by release of springs 28.

Various modifications of the apparatus herein will occur to those skilled in the art and it is intended that the specific embodiment of the invention hereinabove set forth be regarded as illustrative rather than limiting, except as set forth in the claims hereinafter.

I claim:

1. In a printing device a rotary mounted cylindrical printing die having a segment cut axially from the periphery thereof, said cutaway portion acting as a stop to terminate rolling printing action of said die in contact with a surface to be printed, at least one circular spacer disc rotatably mounted co-axial with said printing die, said disc being of larger diameter than said printing die and having a segment cut through the circular periphery to define a peripheral portion discontinuing the regular arcuate portion contiguous therewith, whereby said disc will terminate its rolling action when the cut-away portion bears against a flat surface, thereby acting as a rotary stop for the disc in contact with the surface capable of receiving print, said cut segmental portion of said disc being of lesser diameter than the outer periphery of said printing die, whereby said printing die is capable of imparting print only when the cut segmental portion of the spacer disc is adjacent the surface to be printed.

2. In a printing device a rotary mounted cylindrical printing die having a segment cut axially from the periphery thereof, said cutaway portion acting as a stop to terminate rolling printing action of said die in contact with a surface to be printed, annular spacer discs rotatably mounted at each end and co-axial with said printing die, said discs being of larger diameter than said printing die and having simi-

larly cut segments from the peripheries thereof to effect a rotary stopping action in contact with the surface capable of receiving print, said cut segmental portion of said discs being of lesser diameter than the outer periphery of said printing die, whereby said printing die is capable of imparting print only when the cut segmental portion is adjacent the surface to be printed, said printing cylinder and at least one of said spacer discs having cooperative stop projections contacting each other for unitary rotary motion of both the printing cylinder and spacer elements at only one point of their arc of rotation whereby continuous rotation of either rotary unit through a sufficient arc will engage the other unit to bind both for rotation together, said cylindrical printing element and said spacer element, co-axially therewith, being slidably mounted in a frame with both the periphery and cutaway portions thereof extending outwardly thereof for continuous contact with a moving printable surface and means biasing said printing cylinder outwardly of said frame for imparting resilient pressure of said printing element against the surface to be printed.

3. Printing device having a cylindrical rotary printing die mounted therein for printing by a frictional rolling contact with the surface to be printed, a segment cut axially through the periphery of said cylinder defining a discontinuous region acting as a stop against the surface to be printed whereby rotary action is discontinued, a pair of rotatably mounted annular spacer members at each end thereof co-axial therewith for substantially independent rotation, said annular members having a greater diameter than said printing cylinder, said annular members having aligned cutaway segments in the peripheries thereof similarly defining a flat portion tending to discontinue rotary motion in contact with a flat surface, said cutaway segments of said spacer members extending sufficiently deep from the peripheries thereof to define a portion a shorter distance from the axis thereof than the periphery of the printing cylinder whereby when the surface to be printed is in contact with the cutaway portion of said annular members the periphery of said cylindrical printing member will bear against the surface to be printed, means interconnecting both the printing cylinder and said annular members at only one point of their 360 degree arc of rotation, and means for rotating the said arcuate peripheries into contact with the surface to be printed.

4. A printing device adapted to be mounted beside a conveyor for imparting a single imprint upon packages carried thereon, said printing device having a cylindrical rotary printing die mounted therein in position for printing by rolling contact with the surface of a package moving by on said conveyor, a segment cut axially through the periphery of said cylindrical printing die defining a discontinuous region of the cylindrical surface thereof acting as a stop against the surface to be printed whereby rotary action is discontinued in contact therewith, a trip arm extending into the path of a moving package on said conveyor and adapted to rotate said printing cylinder to a degree sufficient to bring the printing surface into rolling contact with the printable surface of the package moving on said conveyor.

5. A printing device adapted to be mounted beside a conveyor for imparting a single imprint upon a package carried thereon, said printing device having a cylindrical rotary printing die mounted thereon in position for printing by rolling contact with the surface of a package moving by on said conveyor, a segment cut axially through the periphery of said cylindrical printing die defining a discontinuous region of the cylindrical surface of said printing die acting as a stop against the surface to be printed whereby rotation thereof is discontinued in contact therewith, spacer means mounted on said printing cylinder adapted to hold the surface thereof out of printing contact with the surface to be printed for a predetermined distance of movement of said package, a trip arm extending into the path of the moving package on said conveyor and adapted to rotate said printing cylinder and actuate said spacer means to a degree sufficient to bring the arcuate portions of the printing element surface into spaced rolling contact with a printable surface of the package moving on said conveyor.

6. A printing device adapted to be mounted beside a conveyor for imparting a single imprint upon a package carried thereon, said printing device having a cylindrical rotary printing die mounted therein in position for printing by rolling contact with the surface of a package moving by on said conveyor, a segment cut axially through the periphery of said cylindrical printing die defining a discontinuous region of the cylindrical surface of said printing die acting as a stop against the surface to be printed whereby rotation thereof is discontinued in contact therewith, a pair of rotatably mounted spacer members at each end of said printing cylinder coaxially therewith for substantially independent rotation, said annular members having a greater diameter than said cylindrical printing die, said annular spacer members having aligned cut away segments in the peripheries thereof similarly defining a flat portion acting as a stop to discontinue rotation thereof in contact with the surface to be printed, said cutaway segments of said spacer elements extending sufficiently deep from the surface thereof to define a portion a shorter distance from the axis thereof than the periphery of the printing cylinder whereby when the surface to be printed is in contact with the cutaway portion of said annular members the periphery of said cylindrical printing member will bear against the surface to be printed, means interconnecting both the printing cylinder and said annular members at only one point of their 360 degrees arc of rotation, a trip arm extending into the path of a moving package on said conveyor and adapted to rotate said printing cylinder and spacer means to a degree sufficient to bring the arcuate portions of the spacer elements and printing die into rolling contact with the printable surface of the package moving on said conveyor.

7. A printing device adapted to be mounted beside a conveyor for imparting a single imprint upon a package carried thereon, said printing device having a cylindrical rotary printing die mounted therein in position for printing by rolling contact with the surface of a package moving by on said conveyor, a segment cut axially through the periphery of said cylindrical printing die defining a discontinuous region of the cylindrical surface of said printing die acting as a stop against the surface to be printed whereby rotation thereof is discontinued in contact therewith, a pair of rotatably mounted spacer members at each end of said printing cylinder co-

axially therewith for substantially independent rotation, said annular members having a greater diameter than said cylindrical printing die, said annular spacer members having aligned cutaway segments in the peripheries thereof similarly defining a flat portion acting as a stop to discontinue rotation thereof in contact with the surface to be printed, said cutaway segments of said spacer elements extending sufficiently deep from the surface thereof to define a portion a shorter distance from the axis thereof than the periphery of the printing cylinder whereby when the surface to be printed is in contact with the cutaway portion of said annular members the periphery of said cylindrical printing member will bear against the surface to be printed, means interconnecting both the printing cylinder and said annular members at only one point of their 360 degrees arc of rotation, said cylindrical printing element and said spacer elements, coaxial therewith, being slidably mounted in a frame with all peripheries and cutaway segmental portions thereof extending outwardly of said frame for continuous contact with the moving printable surface of the package and means resiliently biasing said printing and spacer elements outwardly of said frame for imparting resilient pressure of said elements against the moving surface of the package to be printed for frictionally engaging the same, a trip arm extending into the path of a moving package on said conveyor and adapted to rotate said printing cylinder and spacer means to a degree sufficient to bring the arcuate portions of the spacer elements and printing die into rolling contact with the printable surface of the package moving on said conveyor.

8. Printing device for printing on moving surfaces comprising means actuated by the moving printable surface for setting a rotary printing die in printing relation thereto and means actuated by the same moving surface for delaying the printing action to impart print to such surface at a preselected position thereon, said rotary printing die being resiliently biased against said printable surface and engaged and rotated by movement in frictional contact therewith to impart print to said moving surface, said rotary printing die having a segment cut axially from the periphery thereof whereby the cut-away portion acts as a stop to terminate the rotary printing action of said die.

9. The printing device as defined in claim 8 wherein the means actuated by the moving surface for delaying printing action comprises a trip arm extending outwardly of said printing device adapted to be engaged by the leading edge of an article comprising said moving surface, said trip arm pivotally coacting with said printing die to rotate the same to print imparting position against said moving surface.

10. Printing device adapted to impart a single print to a preselected position on the surface of an article moving on a conveyor positioned beside said printing device, said printing device comprising a rotary printing element having a segment cut axially from the periphery thereof, said cut-away portion acting as a stop to terminate rotary printing action of said printing element in frictional rolling contact with the moving surface to be printed, a spacing element delaying operation of said printing element for a preselected distance from the edge of the moving surface of the article on which printing is to be effected, and a trip arm extending into the path of said moving article, the said trip arm being pivotally mounted on said printing device to rotate said rotary printing element to a position wherein the arcuate peripheral portion is in printing alignment with said moving surface, the arcuate movement of the outer extending portion of said trip arm effecting a delay of the printing action for a predetermined distance from the edge of the article engaging said trip arm.

11. Printing device as set forth in claim 10 wherein the spacing element is adjustable to vary the spacing of the print with respect to the surface to be printed upon.

12. Printing device as set forth in claim 10 wherein the rotary printing die is arcuate and has mounted for rotation in continuous contact therewith a pair of inking rollers.

13. Printing device as set forth in claim 10 wherein both the rotary printing die and spacing element are slidably mounted in a frame and radially biased into resilient contact with the article to be printed upon.

14. The combination of a rotary printing die with a rotary spacing element, said spacing element being adapted to rotate in frictional rolling contact with a relatively moving surface to be printed upon and to hold said printing die out of rotary printing engagement therewith whereby said printing die is inoperative only for a preselected space measured by said spacing element prior to actuation of said printing die against the relatively moving surface upon which print is to be applied.

15. Printing device comprising a rotary cylindrical printing die having a spacing element operative to delay printing action of said rotary printing element, said spacing element comprising a circular member mounted coaxially with said rotary printing element adapted to be rotated by a surface for receiving print moving relative thereto, said spacing element being of larger diameter than said printing element and having a segment cut away from the periphery thereof to be of lesser diameter at the cutaway portion than the diameter of said printing element.

16. In a conveyor marker, a rotary mounted cylindrical printing die having a segment cut axially from the periphery thereof, said cut-away portion forming a relatively flat surface stop to terminate rotation in contact with the moving surface to be printed, said cylindrical printing element being mounted to slide radially in a frame with both the arcuate periphery and cut-away portions thereof extending outwardly thereof for continuous contact with a moving printable surface, means biasing said printing cylinder outwardly of said frame for imparting resilient pressure of said printing element against the surface upon which print is to be imparted, a trip arm pivotally mounted on said frame having one end thereof extending outwardly into the path of a moving article upon a conveyor positioned beside said marker, upon which print is to be imparted, the outwardly extending end of said trip arm being adapted to be engaged by an edge of said moving article for pivotal rotation thereby, the inner end of said trip arm being linked to said rotary printing die to impart rotation thereto by the pivotal movement of the trip arm, whereby the cylindrical portion of said printing die may frictionally engage said moving article for imparting print thereto.

17. Printing device adapted to be mounted beside a conveyor for imparting a single print in a

preselected position on the surface of an article moving by the printing device on said conveyor, said printing device comprising a frame having a rotary printing die mounted for rotation therein and resiliently biased outwardly thereof, said rotary die being generally cylindrical in configuration and having a portion thereof cut away axially thereof to form a relatively flat surface contiguous with the cylindrical surface thereof, said rotary printing die being adapted to rotate by frictional engagement of the cylindrical surface portion thereof with the surface of a moving article upon which print is to be imparted, the relatively flat surface contiguous with the cylindrical portion of said printing die acting as a stop against which the surface to be printed slides to terminate the rotary movement of the printing die in contact said moving article, and a trip arm pivotally mounted on said frame having one end linked to a ratchet and pawl mechanism mounted upon one end of said rotary printing element whereby pivotal movement of said trip arm will rotate said printing element through a short arc to print imparting position, the other

end of said trip arm extending outwardly of said printing device and adapted to be engaged by an edge of the moving article upon which print is to be imparted, said trip arm upon rotation upon its pivot by the passage of said article engaging the same effecting a spacing of the print upon said moving surface.

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