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HORGAN(10) **Pub. No.: US 2017/0184234 A1**(43) **Pub. Date: Jun. 29, 2017**(54) **SYSTEMS AND METHODS FOR HINGE COUPLINGS****Publication Classification**(71) Applicant: **Tyco Fire Products LP**, Lansdale, PA (US)(72) Inventor: **Michael W. HORGAN**, East Greenwich, RI (US)(21) Appl. No.: **15/415,412**(22) Filed: **Jan. 25, 2017****Related U.S. Application Data**

(63) Continuation of application No. 14/879,961, filed on Oct. 9, 2015, now Pat. No. 9,556,985, which is a continuation of application No. 13/504,102, filed on Jun. 25, 2012, now Pat. No. 9,169,952, filed as application No. PCT/US2010/054123 on Oct. 26, 2010.

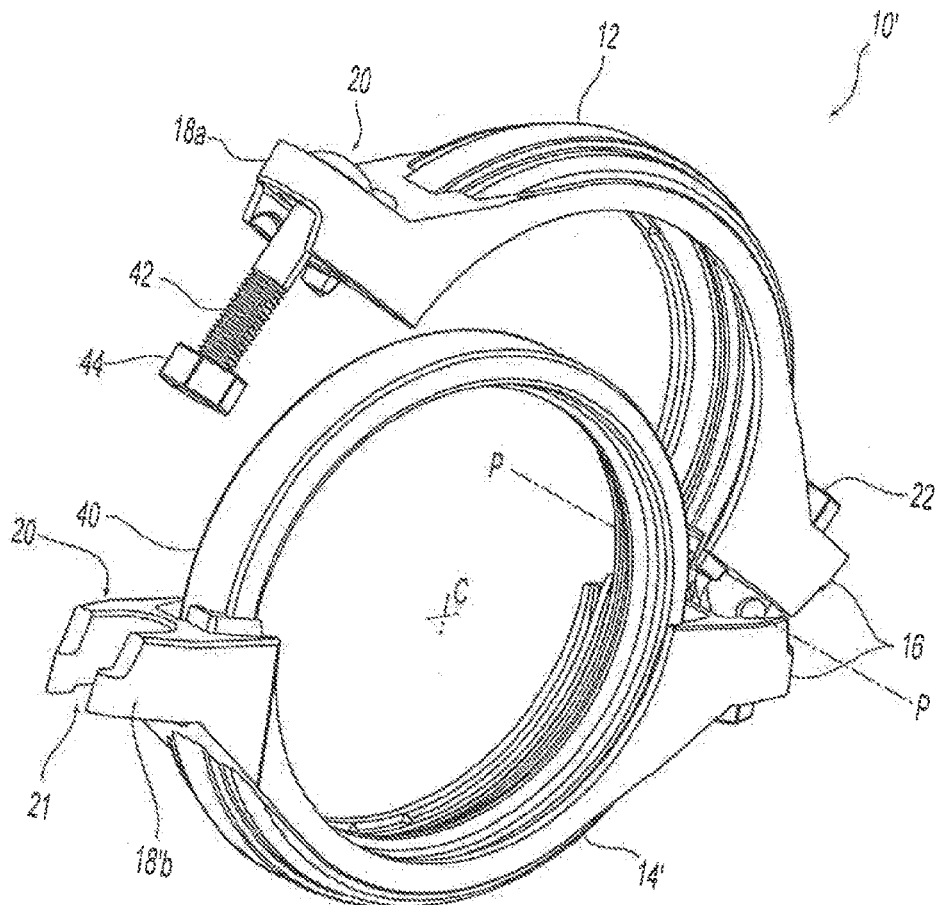
(60) Provisional application No. 61/255,351, filed on Oct. 27, 2009.

(51) **Int. Cl.****F16L 23/10** (2006.01)**F16L 37/124** (2006.01)**F16L 17/04** (2006.01)(52) **U.S. Cl.**CPC **F16L 23/10** (2013.01); **F16L 17/04** (2013.01); **F16L 37/124** (2013.01)

(57)

ABSTRACT

A coupling for coupling pipe segments. The coupling includes a first housing component, a second housing component, and a fastener coupling the first and second components together. The fastener has an aligned configuration defining an axis of alignment such that first and second housing components are in a closed configuration to define a central axis of the coupling. The fastener has a skewed configuration to define a pivot axis of the fastener such that the first and second housing components are in an open configuration. The pivot axis is substantially parallel to the central axis and substantially perpendicular to the axis of alignment.



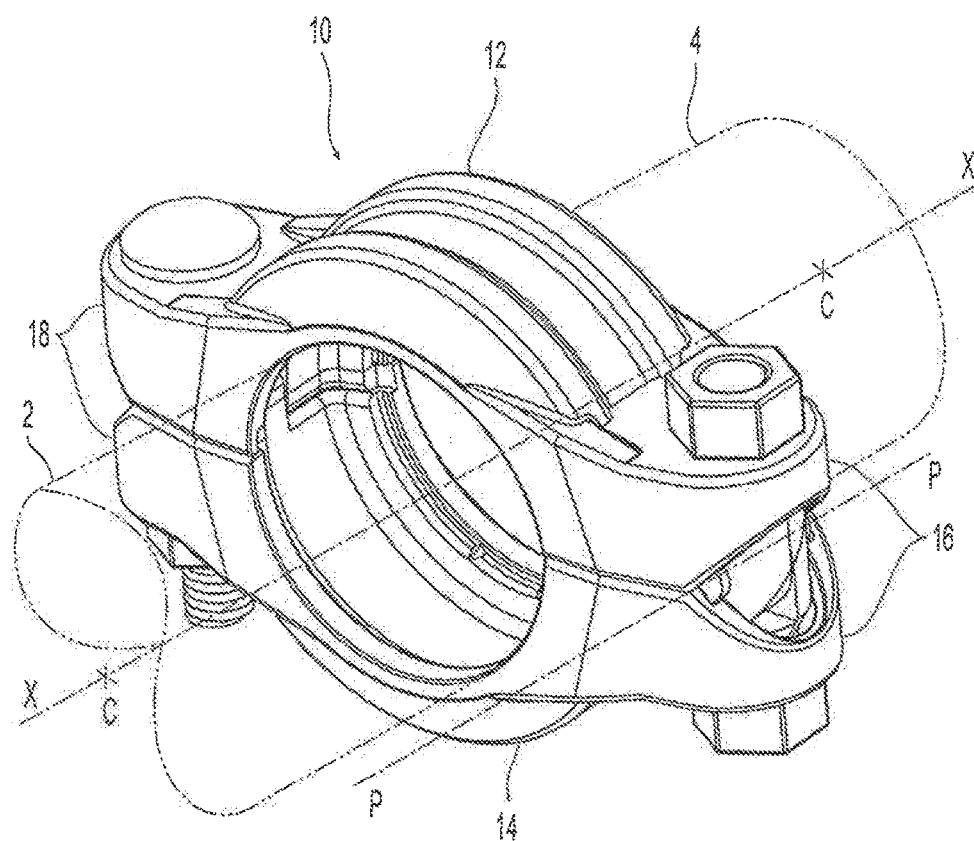


Fig. 1

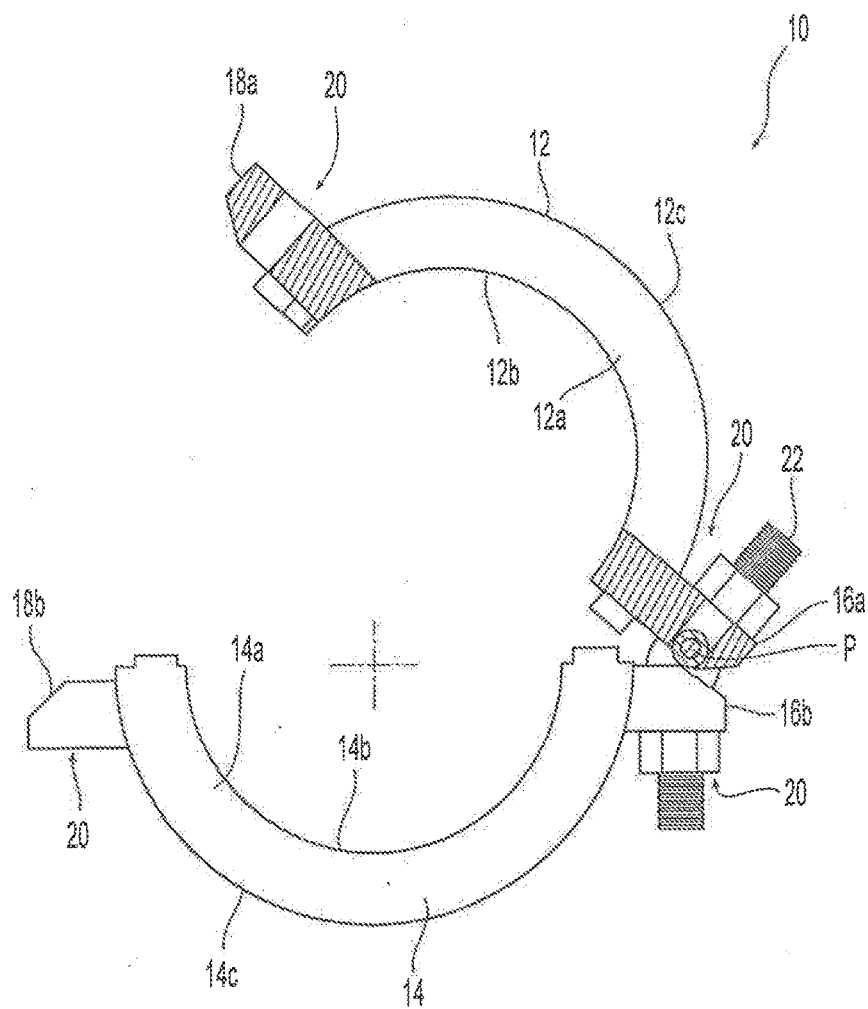


Fig. 2

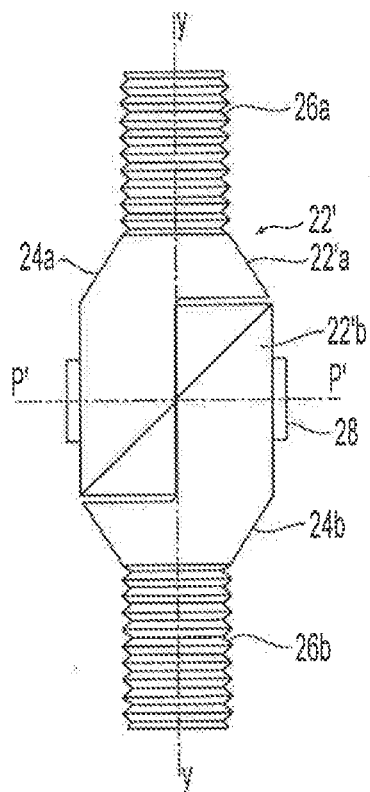


Fig. 3A

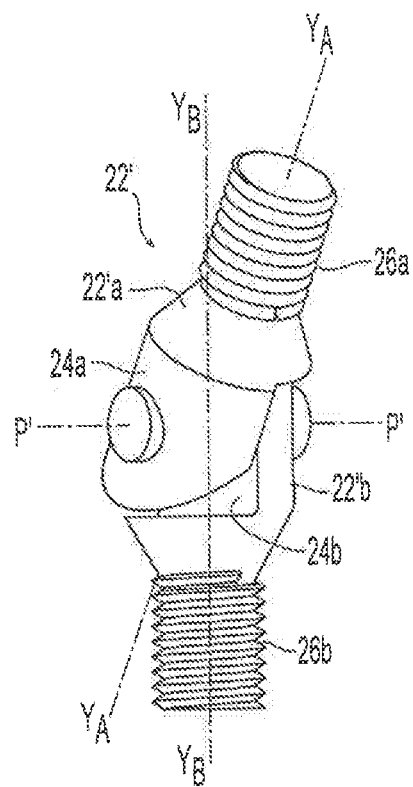


Fig. 3B

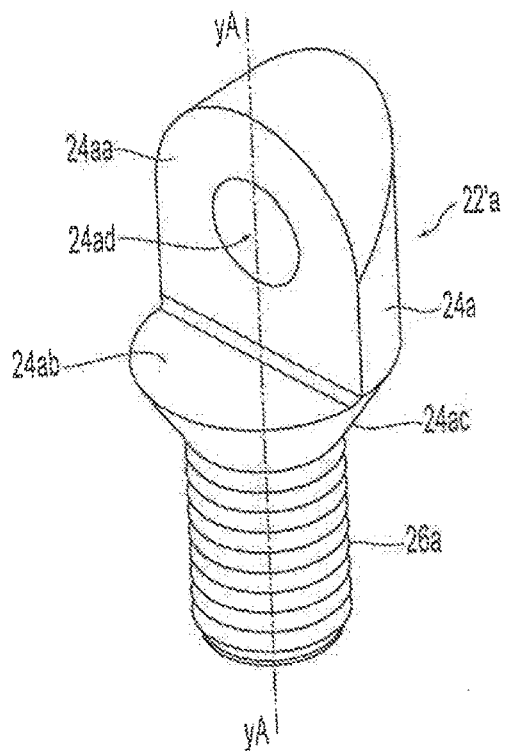


Fig. 3C

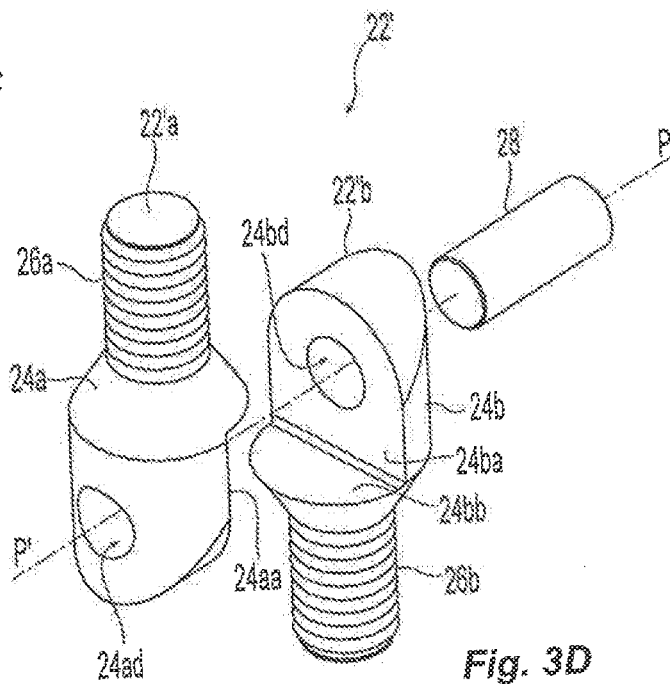
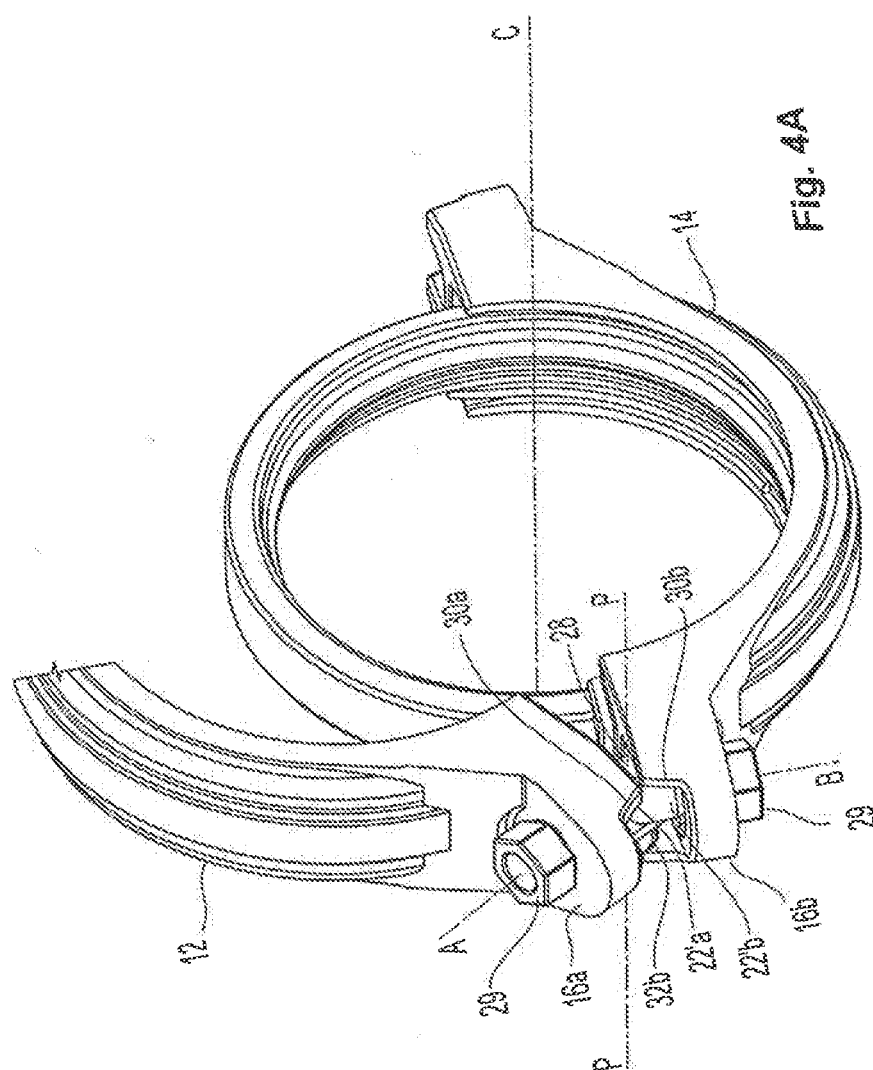
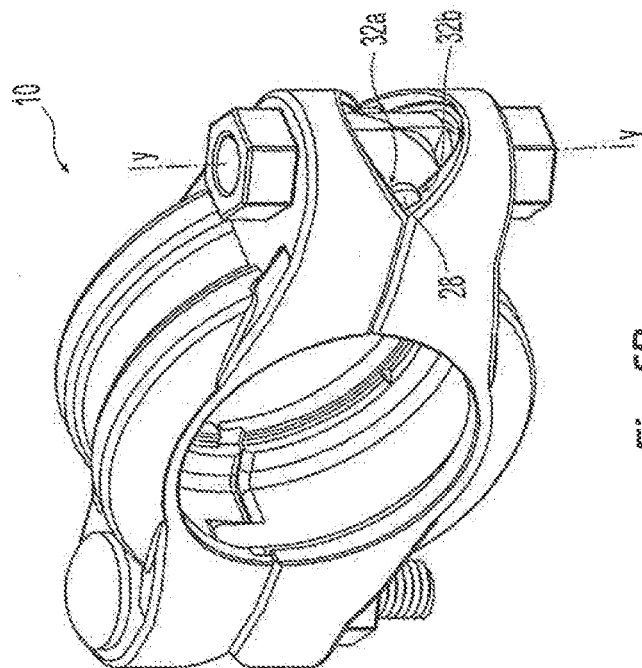
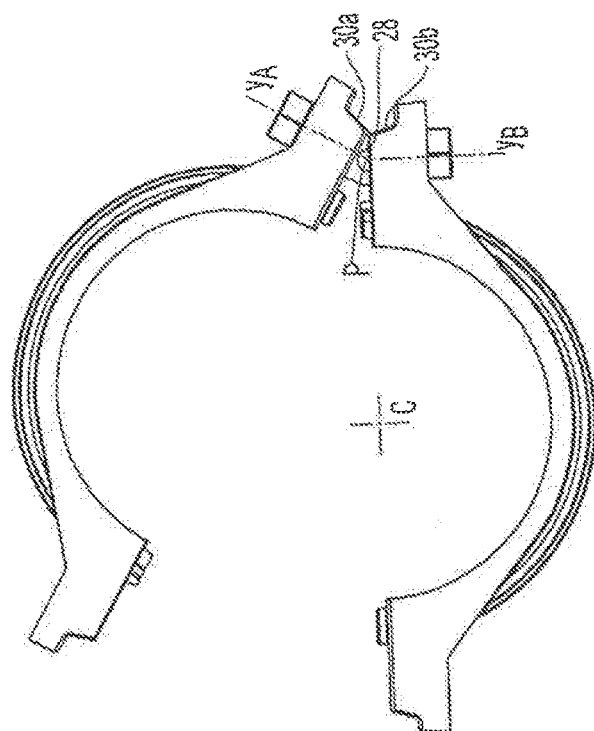


Fig. 3D





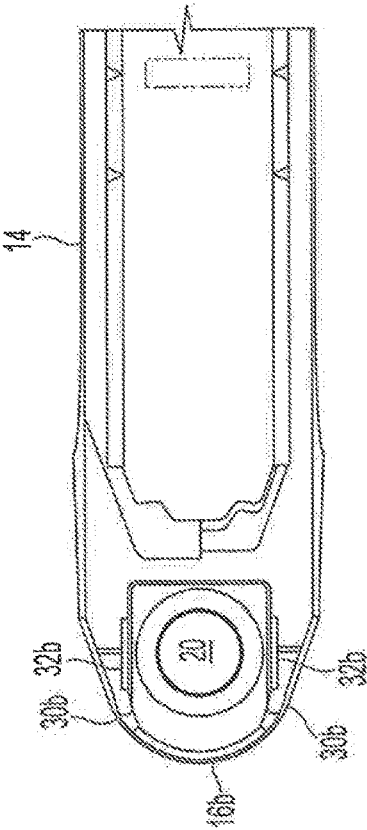


Fig. 6A

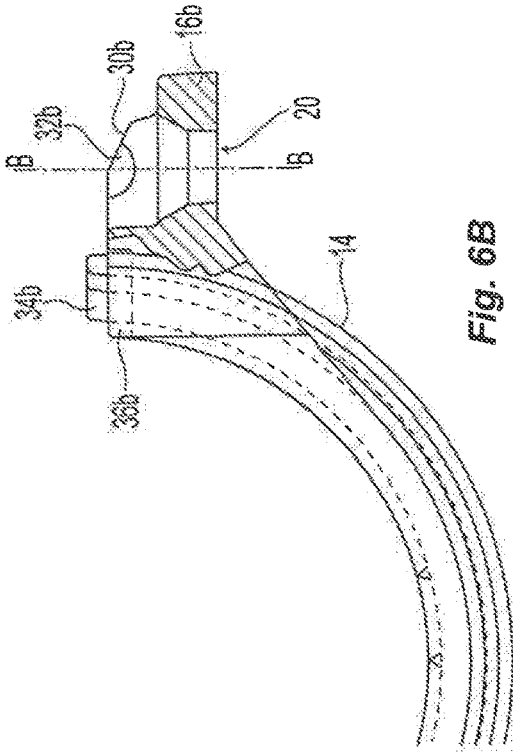


Fig. 6B

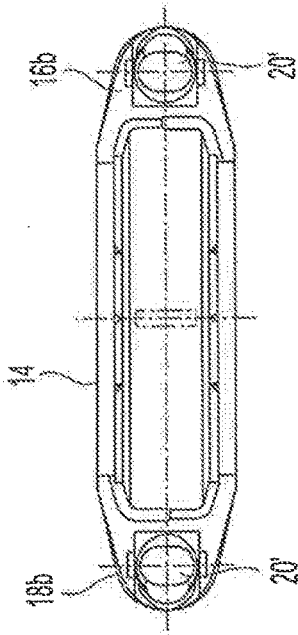


Fig. 6C

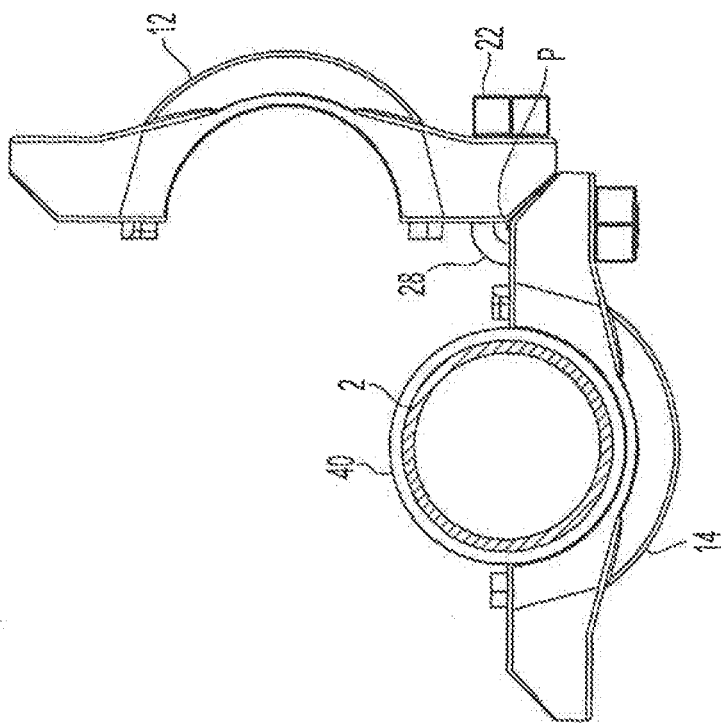


Fig. 7A

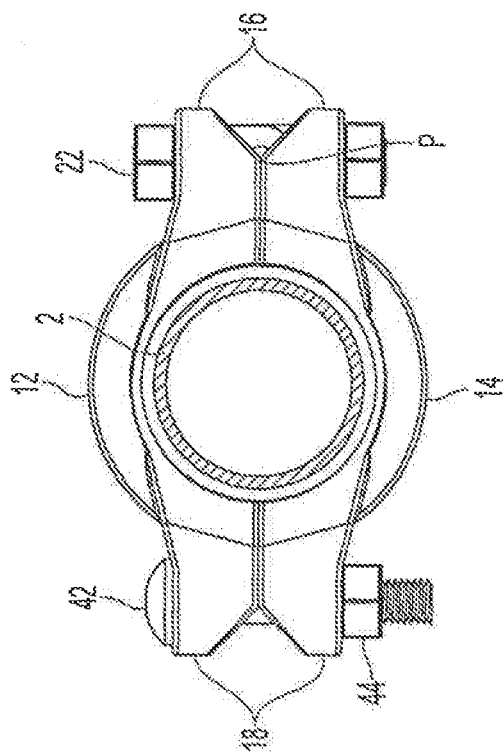


Fig. 7B

Fig. 8A

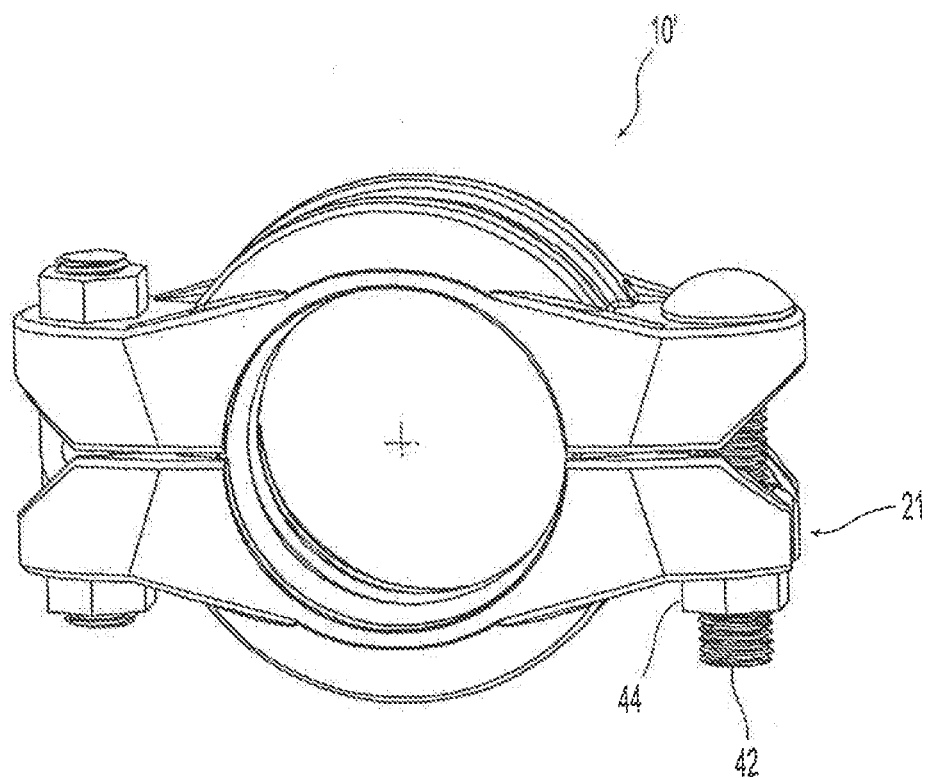
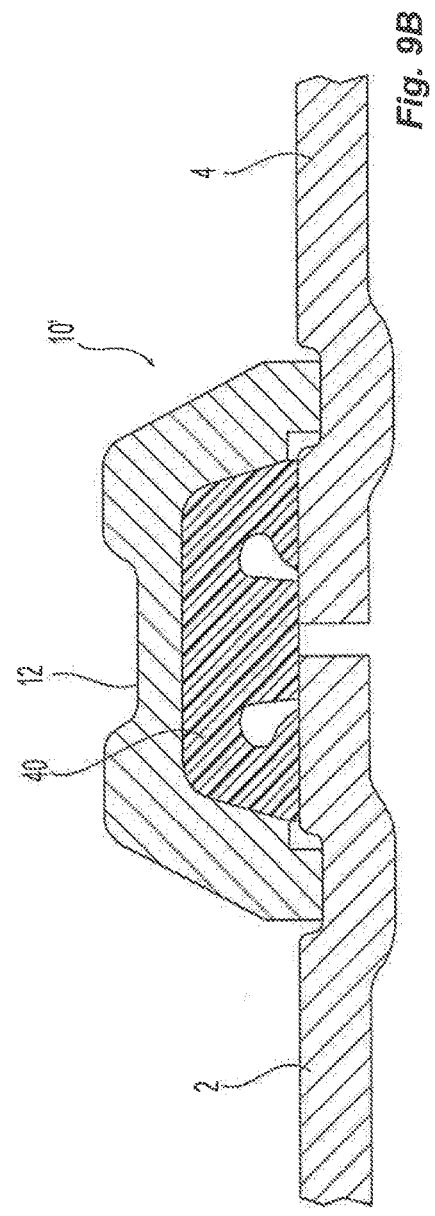
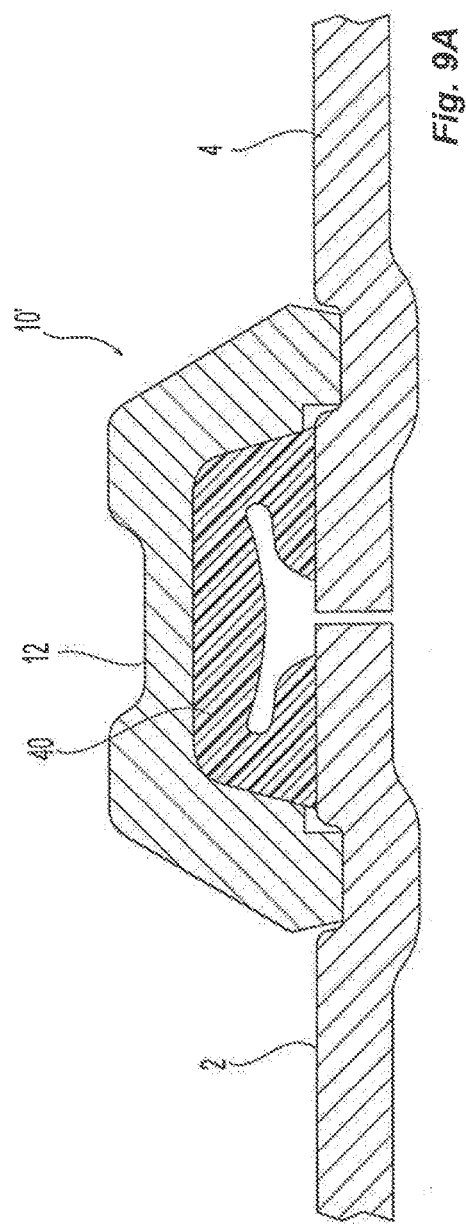
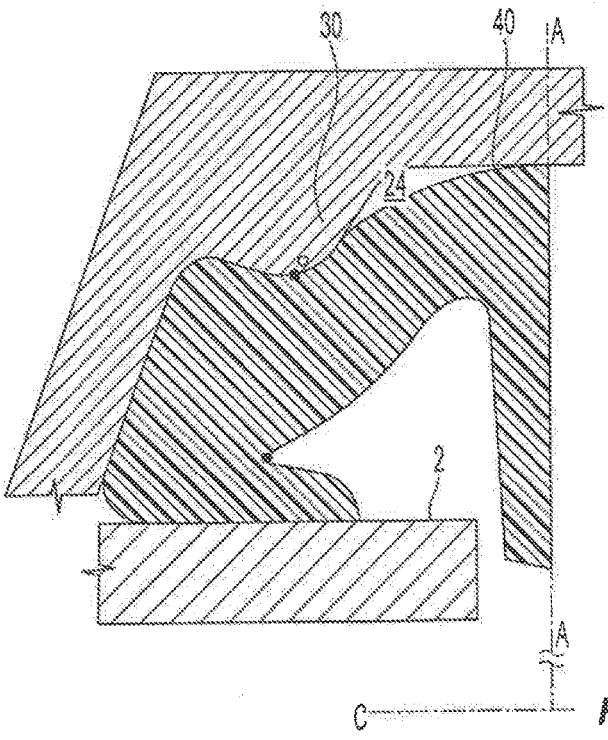
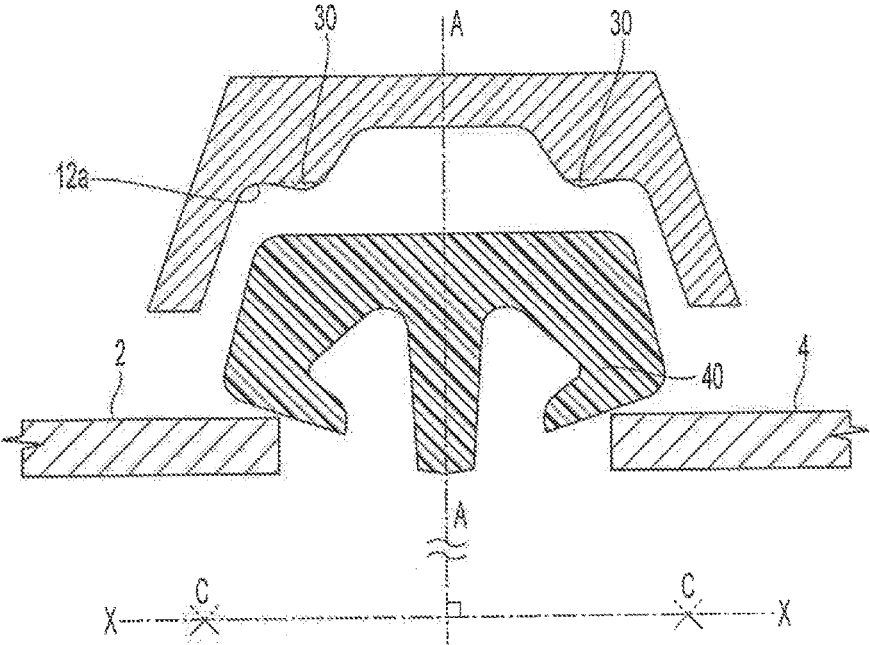


Fig. 8B





SYSTEMS AND METHODS FOR HINGE COUPLINGS

PRIORITY DATA & INCORPORATION BY REFERENCE

[0001] This international application claims the benefit of priority to U.S. Provisional Application No. 61/255,351, filed Oct. 27, 2009, entitled "Systems and Methods for Hinge Couplings" and which is incorporated by reference in its entirety.

TECHNICAL FIELD

[0002] This invention relates generally to pipe fittings and more specifically devices and methods for coupling fluid conveying piping or tubing.

SUMMARY OF THE INVENTION

[0003] Provided are preferred systems and methods for a hinged coupling. The preferred coupling for coupling two pipe segments together includes a first housing component having a first end, a second end, and an arcuate surface extending between the first and second ends of the first component. A second housing component having a first end, a second end, and an arcuate surface extends between the first and second ends of the first component. Each of the first ends of the first and second housing components having a through hole and a fastener disposed within the through holes. The preferred fastener has a first element and a second element to define a first configuration and a second configuration of the fastener. The first configuration of the fastener includes the first and second elements of the fastener being substantially aligned such that the through holes of the first ends of the first and second components are substantially aligned so as to define an axis of alignment extending through the through holes. The second configuration of the fastener comprises the first and second elements being skewed with respect to one another about a pivot axis that extends substantially perpendicularly to the axis of alignment.

[0004] A method is provided for assembling a pipe coupling that includes pivoting about a pivot axis a first fastener element disposed in a first end of a first housing component of the coupling relative to a second fastener element disposed in a first end of a second housing component of the coupling such that the first and second elements are disposed along a linear axis that is substantially perpendicular to the pivot axis so to bring interior surfaces of the first and second housing components opposed to one another to define a central axis of the coupling that runs substantially parallel to the pivot axis. The method further includes securing a second end of the first housing component to the second end of the second housing component.

[0005] In another preferred embodiment of a coupling for coupling pipe segments, the coupling includes a first housing component, a second housing component, and a fastener coupling the first and second components together. The fastener has a first element and a second element defining an aligned configuration including an axis of alignment such that first and second housing components are in a closed configuration so as to define a central axis of the coupling, the first and second elements of the fastener having a skewed configuration to define a pivot axis of the fastener such that the first and second housing components are in an open

configuration. The pivot axis is substantially parallel to the central axis and substantially perpendicular to the axis of alignment.

[0006] In another preferred embodiment, a coupling for coupling pipe segments includes a first housing component and a second housing component. The first and second housing components have an open configuration and a closed configuration to define a central axis of the coupling. A fastener couples the first and second components together, in which the fastener has a first element engaged with the first housing component along a first engagement axis and a second element engaged with the second housing component along a second engagement axis. The first element is coupled to the second element such that the first and second elements of the fastener pivot with respect to one another about a pivot axis that is perpendicular to a plane defined by at least one of the first and second engagement axes. The pivot axis is substantially parallel to the central axis of the coupling and defines the pivot axis about which the first and second housing components pivot relative to one another between the open and closed configuration.

[0007] In one preferred embodiment of a coupling, the coupling includes a first housing component having a first through hole and a second housing component having a second through hole. A fastener having a first fastener element is engaged with the first housing component and a second fastener element engaged with the second housing component, the fastener coupling the first and second components together so that the first and second housing components pivot with respect to one another. Preferably the fastener has a cast fit within the through holes.

[0008] In yet another embodiment, a fastener is provided that includes a first element and a second element, each of the first and second elements having an enlarged head portion and a shank portion depending from the head portion along a longitudinal axis.

[0009] The head portion of each of the first and second elements including a through hole to define a pivot axis. A pin disposed in the through holes of each head portion of the fastener to couple the first element to the second element such that the head portions of the first and second elements pivot with respect to one another about the axis.

BRIEF DESCRIPTIONS OF THE DRAWINGS

[0010] The accompanying drawings, which are incorporated herein and constitute part of this specification, illustrate exemplary embodiments of the invention, and, together with the description given above, serve to explain the features of the invention.

[0011] FIG. 1 is a perspective view of a preferred coupling.

[0012] FIG. 2 is a partial cross-sectional view of the coupling of FIG. 1.

[0013] FIG. 3A is a preferred embodiment of a fastener for use in the coupling in FIG. 1.

[0014] FIG. 3B is a perspective view of the fastener of FIG. 3A.

[0015] FIG. 3C is a perspective view of an element in the fastener of FIG. 3A.

[0016] FIG. 3D is an exploded view of the fastener of FIG. 3A.

[0017] FIG. 4A is a perspective view of another preferred coupling.

[0018] FIG. 5A is an open configuration view of a preferred coupling.

[0019] FIG. 5B is a perspective view of a preferred coupling in a closed configuration.

[0020] FIG. 6A is a plan view of a preferred bolt pad extension of a coupling housing component.

[0021] FIG. 6B is an elevation view of the preferred bolt pad extension of FIG. 6A.

[0022] FIG. 6C is a plan view of another preferred bolt pad extension of a coupling housing component.

[0023] FIG. 7A is a preferred coupling in an open assembly configuration.

[0024] FIG. 7B is the preferred coupling in FIG. 7A in a closed configuration.

[0025] FIG. 8A is another preferred coupling in an opened configuration.

[0026] FIG. 8B is the coupling of FIG. 8A in a closed configuration.

[0027] FIG. 9A is a partial cross-sectional view of preferred coupling assembly in a pipe joint using C shaped gasket.

[0028] FIG. 9B is a partial cross-sectional view of preferred coupling assembly in a pipe joint using a Tri-seal gasket.

[0029] FIG. 10A is a partial uncompressed schematic view of a preferred coupling housing and gasket arrangement.

[0030] FIG. 10B is a compressed view of the arrangement of FIG. 10A.

DETAILED DESCRIPTION

[0031] Shown in FIG. 1 is a preferred embodiment of a hinged coupling 10 secured about two preferably grooved pipe segments 2, 4 so as to couple the pipe segments 2, 4 together along a pipe axis. The preferred coupling includes two housing components 12, 14 which surround a gasket (hidden) to form a fluid tight seal about the end of the pipe segments 2, 4. More specifically, the preferred coupling 10 includes an upper housing 12 and a lower housing 14 each of which is preferably dimensioned to cradle and house approximately one-half of the pipe assembly. Alternatively, the coupling 10 could include more than two housing components provided adjacent components could be coupled together in a hinged arrangement as described herein.

[0032] The coupling 10 includes a coupled end 16 and a preferably diametrically opposed securement end 18. The coupling 10 is preferably preassembled for the user such that at the coupled end 16, the upper housing 12 is coupled to the lower housing 14 by the manufacturer before delivery to the end user. The housing components 12, 14 additionally pivot with respect to one another about a pivot axis P-P located at the coupled end 16. Accordingly, a user can place the coupling 10 in an open configuration, as seen for example in FIG. 2, locate the lower housing component 14 about the ends of the pipe segments 2, 4 and pivot the upper housing 12 with respect to the lower housing component 14 so as to enclose the ends of the pipe segment therebetween in the closed configuration of FIG. 1. At the securement end 18, the housing components 12, 14 are secured together such that the housing components 12, 14 cannot pivot with respect to one another and a fluid tight seal is formed about the pipe end segments 2, 4.

[0033] Shown in FIG. 2 is the coupling 10 in a partially cross-sectional view. In this embodiment, each of the lower

and upper housing components 12, 14 are similarly configured. More specifically, the upper housing component 12 includes a first end 16a, a second end 18a with a preferably substantially arcuate segment 12a extending between the first and second ends 16a, 18a. Similarly, the lower housing component 14 includes a first end 16b, a second end 18b, with a preferably arcuate segment 14a extending between the first and second ends 16a, 18a. With the housing components in their closed configuration, the arcuate segments are opposed to define a center axis C-C of the coupling 10. Each of the arcuate segments 12a, 14a have an interior surface 12b, 14b defining a gasket cavity for engaging and housing a gasket seal. Each of the arcuate segments 12a, 14a define the outer peripheral surface segment 12c, 14c of the arcuate segments. Preferably, the peripheral surfaces 12c, 14c of the housing components together define an arcuate to substantially circular profile, but other geometric profiles are possible such as polygonal with multiple linear lines.

[0034] Each of the first end 16a, 16b and the second end 18a, 18b of the first and second housing components 12, 14 is preferably defined by a bolt pad extension having a recess and more preferably a through hole 20 for receipt of one end of a fastener to couple the housing components together at each of the coupling end 16 and securement end 18. Shown in FIG. 2 is a preferred pivot fastener 22 engaged or disposed within the through holes 20 of the bolt pads at the first ends 16a, 16b of the upper and lower housing components 12, 14. In addition to coupling the housing components 12, 14 together, the pivot fastener 22 provides for relative pivoting motion between the upper and lower housing components 12, 14 about a pivot axis P that preferably runs parallel to the central axis C of the coupling 10. Because the housing components 12, 14 in this embodiment are identical and preferably symmetrical, the preferred pivot fastener 22 can be installed in either of the two ends 16, 18 of the coupling 10 such that either end of the coupling can serve as the coupling end 16 and the opposite end could serve as the securement end 18. Preferably disposed within the through holes 20 of the second ends 18a, 18b of the housing components 12, 14 would be another fastener, preferably a fixed straight bolt secured by a nut (not shown) in order to maintain the coupling 10 in the closed configuration.

[0035] Shown in FIGS. 3A and 3B is a preferred pivot fastener 22'. The preferred pivot fastener 22' preferably includes a first element 22'a and a second element 22'b coupled or engaged with the first element 22'a such that the elements 22'a, 22'b can pivot with respect to one another to define two or more configurations. For example, shown in FIG. 3A are the first and second elements 22'a, 22'b in a first configuration in which the elements 22'a, 22'b are substantially axially aligned along axis Y-Y, and shown in FIG. 3B the elements 22'a, 22'b are pivoted relative to one another such that first element 22'a extending along axis YA-YA is skewed with respect to the second element 22'b extending along axis YB-YB.

[0036] The first and second elements 22'a, 22'b are preferably substantially identical. Shown in FIG. 3C is a preferred element 22'a. Each of the elements 22'a, 22'b of the fastener have a preferably enlarged head portion 24a and a shank portion 26a depending from the head portion 24a along the longitudinal axis YA-YA of the element 22'a. The head portions 24a, 24b preferably engage or cooperatively operate with one another to define the pivoting relationship

between the first and second elements **22'a**, **22'b**. In one particular preferred embodiment, the head portion **24a** includes a first planar bearing surface **24aa** located in a plane that includes the central longitudinal axis of the element **22'a**. The head portion **24a** further preferably includes a second planar shoulder surface **24ab** that extends perpendicular to the first planar bearing surface **24aa**. Transitioning from the enlarged head portion **24** to the preferably narrower shank portion **26a** of the elements **22'a** is a preferred frustoconical transition portion **24ac**. The transition portion **24ac** can define an alternate geometry such as for example, circular cylindrical or a step transition from the head portion **24a** to the shank portion. The shank portion **26a** is preferably threaded for securing the fastener **22'** within the bolt pad ends **16**, **18** of the coupling housing components **12**, **14**.

[0037] Shown in FIG. 3D is an exploded assembly view of the preferred pivot fastener **22'**. In the assembly, the planar bearing surfaces **24aa**, **24ba** of the elements **22'a**, **22'b** engage one another to define at least a line of contact in the plane that includes the central longitudinal axis of the pivot axis fastener. To secure the two elements together, the preferred fastener **22'** includes a pin element **28** that is disposed within through holes **24ad**, **24bd** formed in each of the head portions **24a**, **24b** of the elements **22'a**, **22'b** to define a pin axis P'-P'. The through holes **24'ad**, **24'bd** preferably extend orthogonally through the first planar bearing surfaces **24aa**, **24ba** in each element **22'a**, **22'b**. The elements **22'a**, **22'b** rotate or pivot with respect to one another about the pin **28** and its axis P'-P'. Referring again to FIG. 3A, the assembled preferred pivot fastener **22'** includes two preferably threaded ends **26a**, **26b** that are opposed about an enlarged central portion **24a**, **24b** along the longitudinal axis in the axially aligned configuration of the pivot fastener **22'**. The central portion **24a**, **24b** preferably defines a substantially circular cylindrical outer surface geometry. Alternatively, the central portion **24a**, **24b** could be rectangular cylindrical or cubical. The pin **28** preferably defines an axial length that is greater than the width of the central portion **24a**, **24b** such that the ends of the pin **28** protrude beyond the through hole openings **24ad**, **24bd** of the respective head portions **24a**, **24b** of the elements **22'a**, **22'b**. Alternatively, the pin **28** could be integrally formed on one element **22'a** to be received in a through hole or recess formed on the other element **22'b**. Further in the alternative, the each of the elements **22'a**, **22'b** can be formed so as to have corresponding structures that engage and cooperate with one another such that the elements **22'a**, **22'b** can pivot with respect to one another in a manner as described herein.

[0038] As described above, the fastener **22** is preferably secured within the through holes **20** of the bolt pads formed at one of the ends **16**, **18** of the housing components **12**, **14** to provide the coupling **10** with a housing in which the components **12**, **14** are coupled together at one end and yet pivot with respect to one another about an axis P-P. Shown in FIG. 4A is another view of the preferred coupling **10** assembly in an open configuration with the preferred the pivot fastener **22'** secured at the coupling end **16**. Each end of the preferred fastener pivot **22'** is secured within the through hole **20** of the bolt pad extensions **16a**, **16b** of the respective upper and lower housing components **12**, **14**. To secure the fastener **22'**, a nut **29** is disposed about the threaded shank **26a**, **26b** of each element **22'a**, **22'b** of the fastener **22'**. With the fastener **22'** installed, the central

portion **24** of the fastener is located between the housing components **12**, **14** such that the pin **28** pivot and its axis of the is disposed parallel to the central axis of the coupling **10** to thereby define the pivot axis for the coupling assembly.

[0039] With the fastener **22'** properly located and disposed within the through holes **20** of the bolt pad extensions end **16a**, **16b**, the housing components can pivot with respect to one another about the pivot axis P-P to go from the open configuration, as shown for example in FIG. 5A to the closed configuration shown in FIG. 5B. In the closed configuration of the coupling **10**, the fastener **22'** is preferably disposed such that the through holes **20** are substantially axially aligned, and in the open configuration, the fastener **22'** is disposed such that the through holes are skewed with respect to one another.

[0040] The range of angles through which the housing component may pivot with respect to one another is preferably only limited by the angular range over which the two elements **22'a**, **22'b** can pivot with respect to one another and the interference between the housing components over that angular range. Referring back to FIG. 3B, the first element **22'a** has a preferred angular range of rotation about 180° degrees and more preferably 210° degrees about the fastener pivot axis P'-P' relative to the second element **22'b**. Accordingly, the upper component **12** preferably pivots through a corresponding range of angular rotation about the pivot axis P-P with respect to the lower housing component **14**.

[0041] To minimize the interference between the housing components and to maximize the relative range of motion, the bolt pad extensions ends that house the pivot fastener **22'** have an angled notch at the outer edge of the bolt pad extension **16a**, **16b** in the area that defines through hole **20**. For example, referring to FIGS. 4A and 5A, the bolt pads taper narrowly at the outer perimeter edge of the housing ends **16a**, **16b** to define the angled surface **30a**, **30b**. The angled surfaces provide for a preferred notch or gap about the pivot axis P-P through which the housing components **12**, **14** can rotate relative to one another without interference.

[0042] In order to facilitate the relative motion between the housing components **12**, **14** and closed sealed configuration of the coupling **10** about the pipe segments, it is desirable for the relative pivot motion between the housing components **12**, **14** to occur in a common plane. Accordingly, the preferred coupling assembly **10** provides that relative rotation between the components about an axis YA-YA, YB-YB extending along a through hole **20** is minimized or more preferably eliminated. In the preferred coupling **10**, the pivot fastener **22'** engages the interior surface of the bolt pad extension end **16a**, **16b** such that the components cannot rotate relative to one another about an axis YA-YA, YB-YB extending through the through hole **20** of the bolt pad extension **16a**, **16b** housing the fastener **22'**. In the assembled coupling of FIG. 4A and as seen in FIG. 5B, the exposed ends of the pin **28** are located within the recesses **32a**, **32b** formed in the interior surface of the bolt pad extensions **16a**, **16b** which define the through holes **20**. Any tendency for the housing components **12**, **14** to rotate relative to one another is minimized or eliminated by the interaction of the exposed ends of the pin **28** and the interior surface defining the recess **32** which hold the exposed ends of the pin **28**.

[0043] Shown in FIGS. 6A and 6B are plan and cross-sectional views of a preferred bolt pad extension. In this

illustrative example, the bolt pad extensions **16b** for lower housing component **14** is being shown, but the opposite bolt pad extension end **18b** can be similarly configured as could the bolt pad extensions **16a**, **18a** of the upper housing component **12**. In the plan view of FIG. 6A, the through hole **20** is shown, and proximate the opening to the through hole **20** are the recesses **32b** diametrically opposed about the opening to engage the exposed ends of the pin **28** in the preferred pivot fastener **22'**. Shown in the cross-sectional view of FIG. 6B, the recess **32b** defines a partially semicircular geometry, although other geometries for the recess **32b** could be chosen provided the recesses were properly located to at least partially house and engage the exposed ends of the pin **28**. With regard to the geometry of the through hole **20**, cross-sectional area of the through hole **20** preferably varies along its axis of elongation B-B or may alternatively be constant provided that the opening can accommodate the desired fastener for coupling the housing components **12**, **14** and/or provide their relative pivot motion about the pivot axis P-P. For example, as shown in FIG. 6C, is another preferred housing component **14** that includes, at both ends **16b**, **18b** through holes **20'** that define an oval geometry to engage a correspondingly shaped portion of a fastener to prevent the fastener from spinning within through hole **20**. Thus, a nut can be threaded about the oval shaped fastener with only one hand.

[0044] Due to the preferred symmetrical configuration of the housing components, as noted above, the preferred coupling pivot fastener **22** can be installed on either end **16**, **18** of the coupling **10**. Moreover, because of the preferred common configuration of the housing components, a single housing component can serve as either the upper housing component or the lower housing component. Having a single symmetric housing component design may be desirable so as to eliminate the need to manufacture or inventory additional housing components for the coupling assembly. Additionally, the single housing component design may eliminate assembly errors by avoiding mismatching and improper assembly of dissimilar parts that require a specific orientation. The housing components **12**, **14** can include additional features to facilitate their assembly, for example as shown in FIG. 6B, the lower housing component **14** may include a tongue **34b** and recess **36b** on each end for mating respectively with a corresponding recess and tongue in the upper housing. The tongue and recesses are preferably located radially inward of the bolt pad through holes **20** relative to the center C of the coupling **10**. Details of the tongue and recess are shown and described in U.S. Pat. No. 6,139,069 which is incorporated by reference in its entirety.

[0045] The preferred use of the pivot fastener can simplify manufacturing of the coupling **10**. Because the preferred pivot fastener **22'** provides for the pivot action of the coupling, and thus the precision fit and tolerances are in the pivot fastener, there is no need to machine the component housings **12**, **14** to form the hinged connection. Accordingly, the coupling **10** can employ a cast fit between the fastener **22'** and the housing components **12**, **14**. As such, the substantial axial alignment or substantial perpendicular orientation between components and elements of the coupling **10** only requires the components to be sufficiently aligned or oriented perpendicular to one another to provide the desired configurations of the coupling.

[0046] Preferably, the coupling **10** is preassembled with the pivot fastener **22** installed and the upper and lower

housing components **12**, **14** coupled together. Shown in FIGS. 7A 7B, is a preferred method of joining pipe or tube segments together using the preferred coupling **10**. A gasket seal **40** is disposed over the ends of the pipe segments **2**, **4** as shown in the end view of FIG. 7A (illustratively shown in cross-section in FIGS. 9A and 9B). With the fastener **22** preferably in the skewed configuration such that the coupling **10** is in the fully open configuration, the lower housing component **14** is brought into engagement with the gasket **40** so as to be received within the housing recess defined by the inner surface **14c** of the lower housing component **14**. The upper housing component **12** is pivoted about the pivot axis P-P, relative to the lower housing component **14** so as to bring the coupling **10** to a closed configuration about the coupling **40** such that the fastener **22** and the through holes **20** in which the fastener is disposed are in the axially aligned configuration. In order to fully seal and form the pipe joint, a separate second fastener **42**, a bolt **42** is inserted and disposed within the axially aligned through holes **20** of the securement end **18** of the coupling **10** opposite the coupled end **16**. A nut **44** is threaded onto the threaded shank of the bolt **42** and secured onto the bolt at an amount ranging from about thirty to about two hundred-fifty foot-pounds (30-250 ft.-lbs) of torque, preferably depending upon the size of the coupling.

[0047] Because the preferred coupling is preassembled, the preferred pivot fastener **22'** is secured or partially secured at its ends by threaded nuts **29**. The nuts **29** are preferably fastened and secured about the threaded shank portions **26a**, **26b** of the fastener **22'** at amount of about 60-100 ft.-lbs, of torque or at an amount of torque to provide a fastened and secure engagement about the threaded shank portions **26a**, **26b**. Alternatively, the ends **26a**, **26b** of the fastener **22'** can be secured to the housing components **12**, **14** by other techniques such as by press fit, staking the ends **26a**, **26b** in place, or using formed ends with other securing structures, i.e. pins, so long as the fastener is properly secured and located within the through holes **20** of the bolt pad extensions **16a**, **16b** and the coupling can satisfy a desired hydrostatic pressure and bending moment rating.

[0048] The above described embodiment of coupling **10** was sealed using a separate fastener **42** that is installed and secured by the end user. In order to provide a more preferably fully pre-assembled coupling device, shown in FIGS. 8A & 8B is an alternate embodiment of the coupling **10'**. In this alternative embodiment the securement end **18** of the coupling **10'** is configured to provide a pre-assembled fastener **42** assembly. The pre-assembled preferred coupling **10'** still includes at its coupled end **16** a fastener **22**, preferably pivot fastener **22'** that couples the upper housing component **12** and lower housing component **14** together and providing a pivot axis P-P about which the component pivots relative to one another. Although the upper housing component **12** is configured as previously described, the lower housing component **14'** preferably includes a slot **21** at the securement end **18**. The slot **21** is in communication with through hole **20** of the bolt pad extension end **18b** and preferably includes a peripheral slot opening located at the outer lateral edge of the housing component **14'**. With the slot opening and slot **21** in the lower housing, a fastener assembly in the form of a combined bolt **42** and nut **44** can be disposed within the through hole **20** of the bolt pad extension end **18a** of the upper housing component **12**. With the slot **21** in communication with the through hole **20**, the bolt pad extension **18b'** provides an elongated or preferably oval shaped, opening to engage a correspondingly shaped portion of the bolt to prevent the bolt spinning within the through hole **20**.

[0049] The pre-assembled coupling 10' eliminates the need for the end user to insert a separate bolt in the securement end 18 of the coupling and secure a nut about the end of the fastener 42. Instead, the end user can locate the coupling 10' about the gasket 40 that is disposed about the aligned pipe segments 2, 4. The coupling is then brought to a closed configuration as shown for example in FIG. 8B. The fastener assembly 42, 44 can be positioned out of the way to allow the housing components to pivot to the closed position. Once in the closed configuration, the fastener assembly 42, 44 can be inserted through the slot 21 and its opening into position in the through hole 20. The nut 44 is then tightened to the desired torque values and the pipe joint can be placed into service. Although the slot 21 and its peripheral opening are described with respect to the lower housing 14', it should be understood that the alternate housing component 14' could be used as the upper housing component of the coupling assembly 10. In such an instance, the fastener assembly 42 can be pre-assembled within the through hole 20 of the lower housing component.

[0050] The above preferred coupling assemblies are preferably configured for joining nominal two inch (2 in.), 2½ inch; three inch (3 in.); four inch (4 in.), six inch (6 in.), or eight inch (8 in.) pipe or tubing, but may be configured to join any size pipe or tubing provided the housing components are configured to hold and secure the pivot fastener 22, in each of the preferred coupling assemblies described above, the pivot fastener 22 in combination with the securement fastener 42 provides sufficient distributed compression force through the housing components about the gasket 40. The inner surfaces 12a, 14a, of the upper and lower housings components 12, 14, which engage the gasket 40, preferably define a profile to maximize and/or maintain a sufficient compressive force against the gasket so as to maintain an effective seal at the pipe joint.

[0051] Shown in FIG. 9A and 9B is a known inner surface 12a of the upper housing component 12 engaged with a preferred gasket 40. Details of an alternative inner surface 12a, 14a and a preferred gasket are shown in the FIGS. 10A and 10B for use in a hinge coupling arrangement described above. Respectively shown in FIGS. 10A and 10B are uncompressed and compressed views in which a tapering inner surface 12a of a coupling housing 12 define a pair of notches 30 which compress lateral sides of a preferred gasket 40 laterally against the interior sidewalls of the housing 12. Detailed description of the gasket and the housing shown in FIGS. 10A and 10B are provided in International Application No. PCT/US10/53970, filed Oct. 25, 2010, entitled "Systems and Methods for Pipe Couplings", and which claims priority to U.S. Provisional Patent Application No. 61/255,409, entitled, "System and Methods for Pipe Couplings," filed Oct. 27, 2009, both which are incorporated by reference. Known gasket configurations may be used with the coupling assemblies described herein. For example, as shown in FIGS. 9A and 9B, standard style "C shaped" or "Tri-seal" gaskets as identified at page 12 in Tyco Fire & Suppression Products Publication IH-1000FP, entitled, "Grinnell®—Grooved Fire Protection Installation Manual" (August 2007) can be used in the preferred coupling assemblies 10, 10'. A copy of page 12 from the installation manual is included in U.S. Provisional Patent Application No. 61/255,351.

[0052] In the preferred coupling 10, 10' and its installation, as described above, the pivot fastener 22 provides at least three functions: i) it couples the housing components 12, 14 together for a pre-assembly that minimize the number of separate components for the end user; ii) the fastener 22 defines the pivot axis of the coupling 10; and iii) in the closed configuration, the fastener 22 provides for a secure hold between the housing components 12, 14 such that the user only has to properly torque one fastener 42 to form a fluid tight seal. The preferred coupling assembly 10 more specifically provides for the preferred fastener 22 that, in its axial configuration, linearly aligns the through holes 20 of the housing components 12, 14 to define the closed configuration of the coupling 10 as seen in FIG. 5B, and in its skewed configuration, defines the open configuration of the coupling 10, as seen in FIG. 5A. Shown in FIGS. 3A-3D and described above is a preferred embodiment of pivot fastener 22'. However alternative configurations of the fastener 22 and its configuration within the components 12, 14 are possible provided that the resultant fastener functions as described. For example, the fastener and the coupling could be configured to hold the coupling in the open and closed configuration with the elements of the fastener in a skewed configuration. Additionally, a fastener could be provided and the bolt pad extensions configured so as not to require the fasteners to extend through a through hole, for example one or more of the bolt pad could include a threaded blind bore for the shanks of the pivot fastener.

[0053] While the present invention has been disclosed with reference to certain embodiments, numerous modifications, alterations, and changes to the described embodiments are possible without departing from the sphere and scope of the present invention. Accordingly, it is intended that the present invention not be limited to the described embodiments, but that it has the full scope defined by the language of the following claims, and equivalents thereof.

1. A coupling for coupling two pipe segments together, the coupling comprising:

- a first housing component having a first end, a second end, and an arcuate surface extending between the first and second ends of the first component;
- a second housing component having a first end, a second end, and an arcuate surface extending between the first and second ends of the first component, each of the first ends of the first and second housing components having a through hole;
- a fastener disposed within the through holes of the first ends of the first and second housing components so as to couple the first and second components, the fastener having a first element and a second element to define a first configuration and a second configuration of the fastener, the first configuration of the fastener includes the first and second elements of the fastener being substantially aligned such that the through holes of the first ends of the first and second components are substantially aligned so as to define an axis of alignment extending through the through holes, the second configuration of the fastener comprises the first and second elements being skewed with respect to one another about a pivot axis that extends substantially perpendicularly to the axis of alignment.

2-33. (canceled)

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