

(12) **United States Patent**  
**Sullivan et al.**

(10) **Patent No.:** **US 10,443,538 B2**  
(45) **Date of Patent:** **\*Oct. 15, 2019**

(54) **COOLING SYSTEM FOR AIR-COOLED ENGINE**

(71) Applicant: **Briggs & Stratton Corporation**,  
Wauwatosa, WI (US)

(72) Inventors: **Ryan A. Sullivan**, West Bend, WI (US); **Gary S. Johnson**, Hales Corners, WI (US)

(73) Assignee: **Briggs & Stratton Corporation**,  
Wauwatosa, WI (US)

(\* ) Notice: Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 174 days.

This patent is subject to a terminal disclaimer.

(21) Appl. No.: **15/352,443**

(22) Filed: **Nov. 15, 2016**

(65) **Prior Publication Data**

US 2017/0058825 A1 Mar. 2, 2017

**Related U.S. Application Data**

(63) Continuation of application No. 14/035,469, filed on Sep. 24, 2013, now Pat. No. 9,500,117.  
(Continued)

(51) **Int. Cl.**  
**F01P 1/00** (2006.01)  
**F02F 7/00** (2006.01)  
(Continued)

(52) **U.S. Cl.**  
CPC ..... **F02F 7/007** (2013.01); **F01P 1/00** (2013.01); **F01P 1/02** (2013.01); **F01P 5/02** (2013.01);  
(Continued)

(58) **Field of Classification Search**  
CPC .... F01P 11/12; F01P 11/14; F01P 1/02; F01P 2001/023; F01P 2001/026; F01P 5/02;  
(Continued)

(56) **References Cited**

U.S. PATENT DOCUMENTS

4,156,409 A \* 5/1979 Nakano ..... F01P 5/06  
123/192.2  
4,841,920 A \* 6/1989 Andreasson ..... F02M 35/06  
123/41.66

(Continued)

FOREIGN PATENT DOCUMENTS

WO WO-2011/068446 A1 6/2011  
WO WO 2011068446 A1 \* 6/2011 ..... F02M 35/024

OTHER PUBLICATIONS

Briggs & Stratton, Commercial Turf Series, Operator's Manual, 2013, 32 pages.  
Briggs & Stratton, Vanguard, Operator's Manual, 2012, 28 pages.

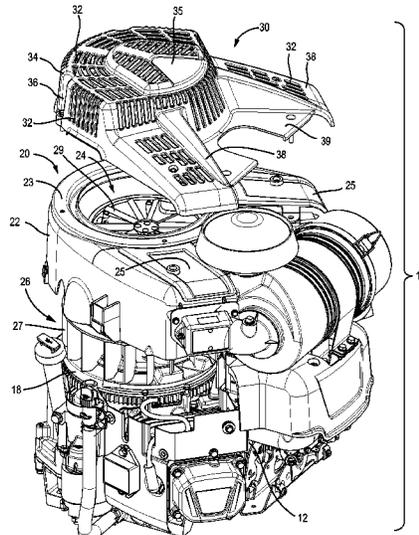
*Primary Examiner* — Grant Moubry

(74) *Attorney, Agent, or Firm* — Foley & Lardner LLP

(57) **ABSTRACT**

An air-cooled internal combustion engine including a crankshaft, a cylinder, a blower assembly including a blower housing and a fan, and a static cover. The static cover includes a main body that is aligned with the crankshaft, an arm that extends from the main body and is aligned with the cylinder, and a plurality of air intake openings. A first subset of the air intake openings is formed through the main body and a second subset of the air intake openings is formed through the arm, and the static cover is configured to prevent user access to a moving component of the engine. The fan is configured to move air into the blower housing through the air intake openings.

**19 Claims, 8 Drawing Sheets**



<b>Related U.S. Application Data</b>					
(60)	Provisional application No. 61/777,947, filed on Mar. 12, 2013.	7,069,896	B2 *	7/2006	Tsunoda ..... F02N 3/02 123/185.3
		7,201,132	B2 *	4/2007	Miyake ..... F01M 1/04 123/195 HC
(51)	<b>Int. Cl.</b>	7,225,765	B2 *	6/2007	Leech ..... F01P 5/06 123/198 E
	<i>F01P 5/02</i> (2006.01)	7,363,885	B2 *	4/2008	Miyake ..... F01P 1/02 123/185.3
	<i>F01P 1/02</i> (2006.01)	7,409,931	B2 *	8/2008	Kochi ..... F01P 1/02 123/41.56
	<i>F01P 11/12</i> (2006.01)	7,412,961	B2 *	8/2008	Doring ..... B27B 17/00 123/198 E
	<i>F01P 5/04</i> (2006.01)	7,475,657	B2 *	1/2009	Dopke ..... F01N 5/02 123/41.56
	<i>F02B 75/00</i> (2006.01)	7,950,355	B2 *	5/2011	Nakamizo ..... F01P 11/12 123/41.67
	<i>F02B 75/22</i> (2006.01)	7,958,855	B2 *	6/2011	Nakamizo ..... F01P 5/06 123/41.65
	<i>F02B 75/18</i> (2006.01)	8,424,498	B2 *	4/2013	Tiefenthaler ..... E01H 1/0809 123/41.63
(52)	<b>U.S. Cl.</b>	8,720,392	B2 *	5/2014	Tiefenthaler ..... E01H 1/0809 123/41.56
	CPC ..... <i>F01P 5/043</i> (2013.01); <i>F01P 11/12</i> (2013.01); <i>F02B 75/007</i> (2013.01); <i>F02B 75/22</i> (2013.01); <i>F02B 2075/1808</i> (2013.01)				
(58)	<b>Field of Classification Search</b>				
	CPC ..... F01P 5/06; F01P 7/02; F02F 7/007; F02F 7/0021; F02F 1/28; F02F 1/04				
	See application file for complete search history.				
(56)	<b>References Cited</b>				
	U.S. PATENT DOCUMENTS				
	5,542,380 A * 8/1996 Chatten ..... B27B 17/00 123/198 E				
		2003/0236595	A1	12/2003	Frankel et al.
		2004/0025827	A1	2/2004	Davis
		2012/0330496	A1	12/2012	Eser et al.
		2014/0261259	A1	9/2014	Sullivan et al.

\* cited by examiner

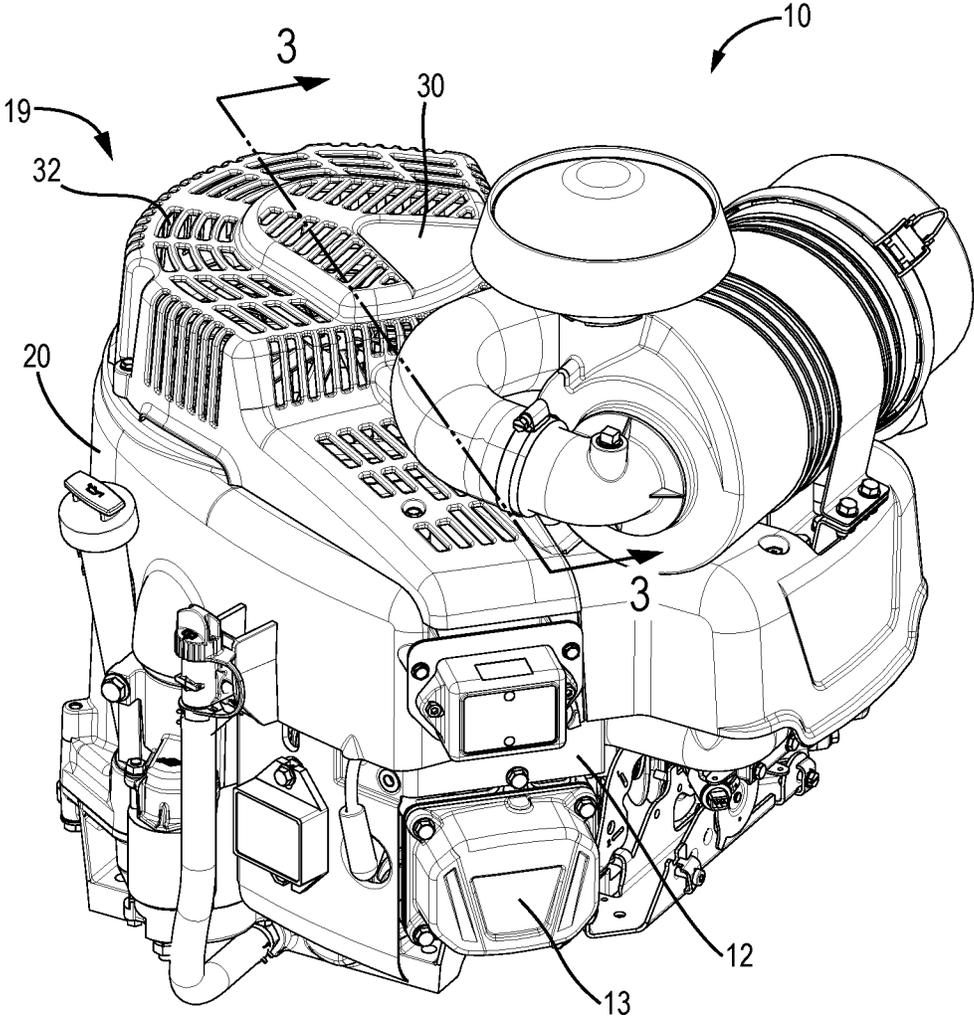


FIG. 1

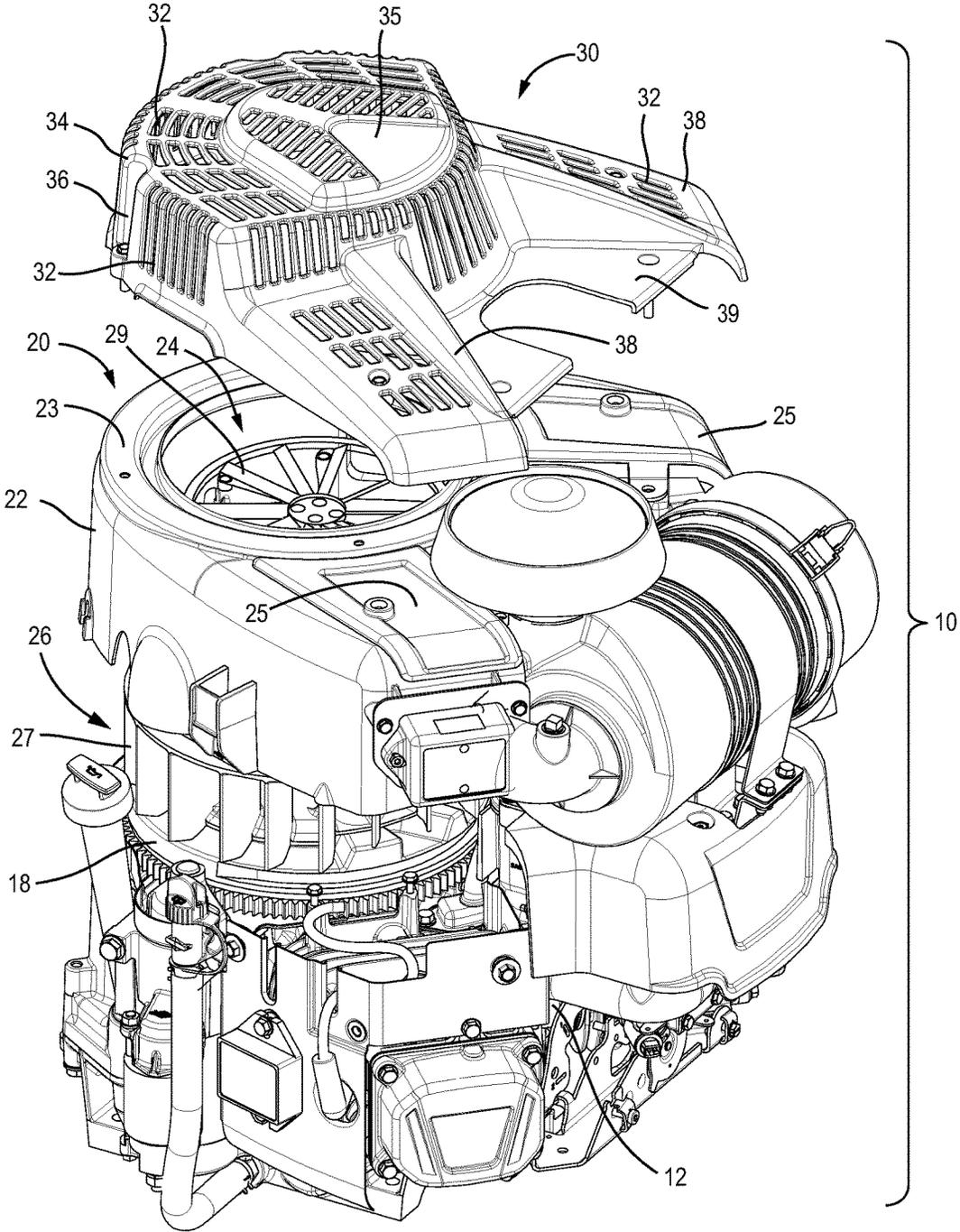


FIG. 2

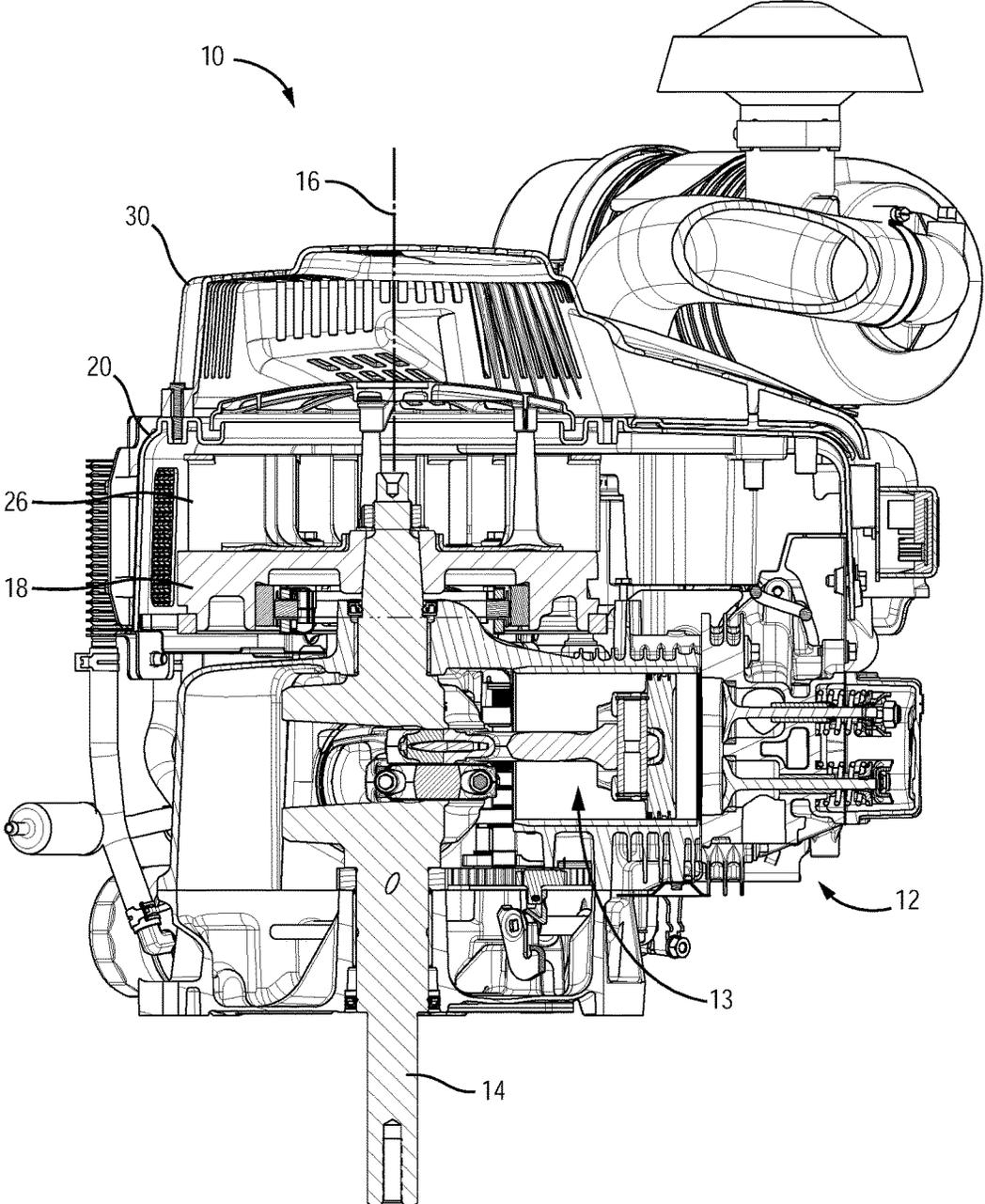


FIG. 3

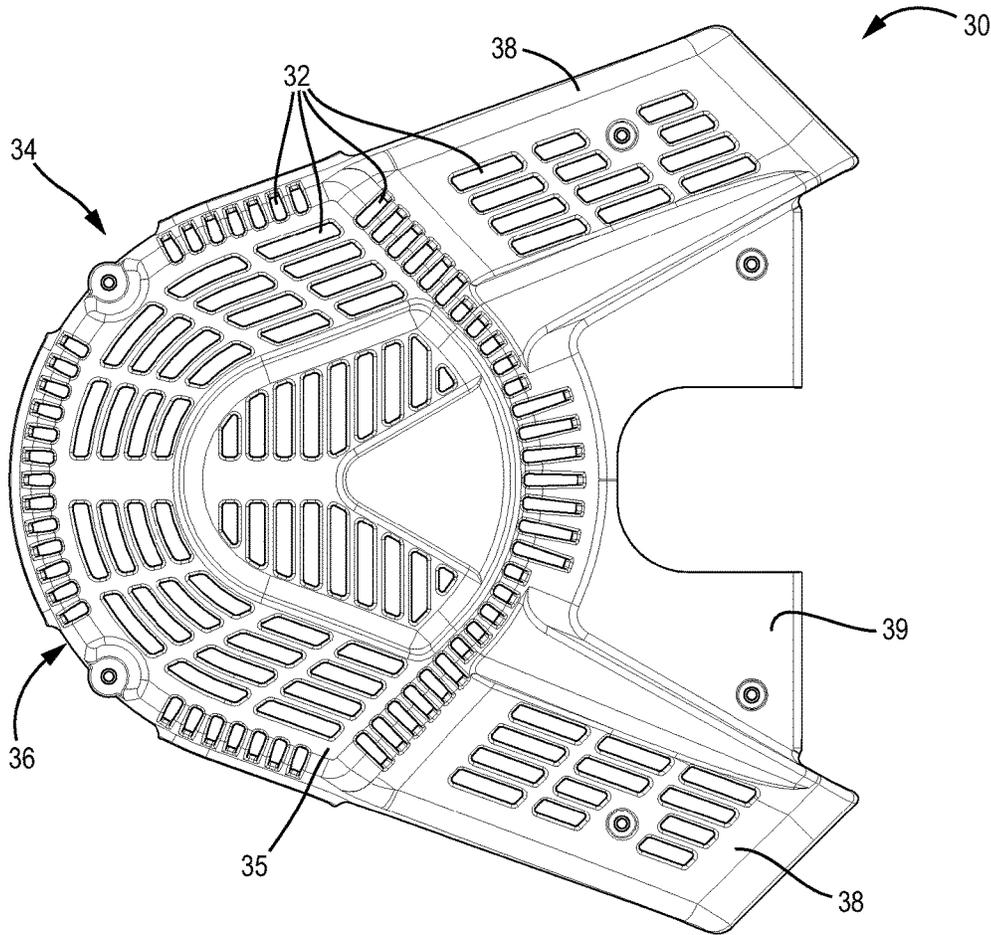


FIG. 4

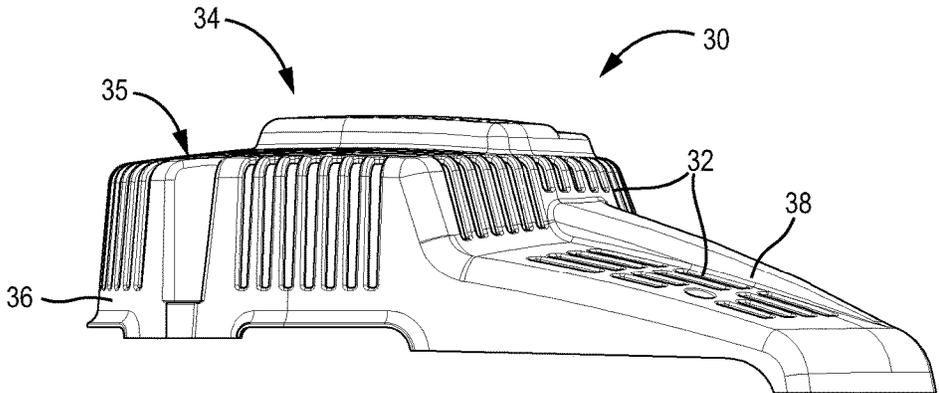


FIG. 5

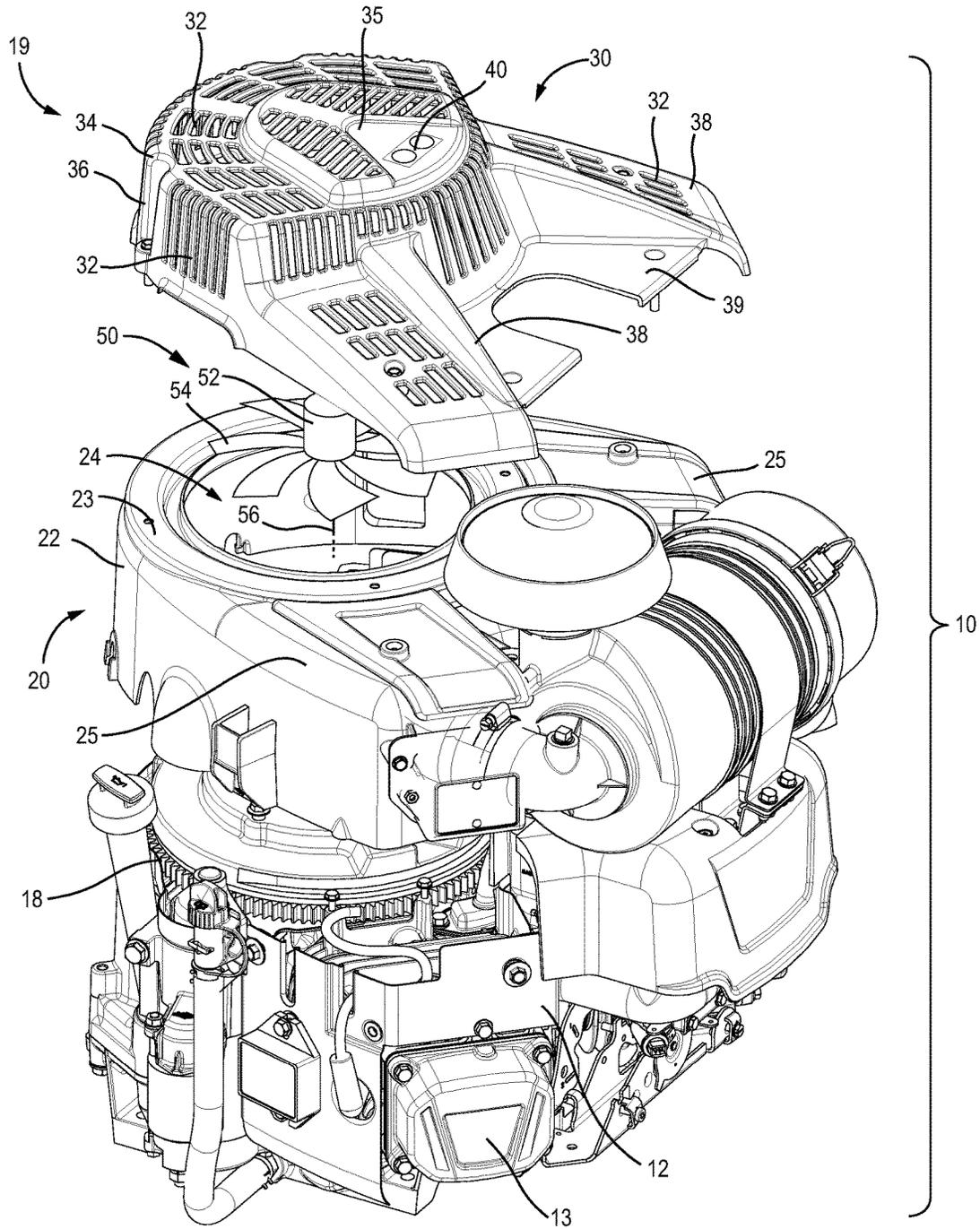


FIG. 6

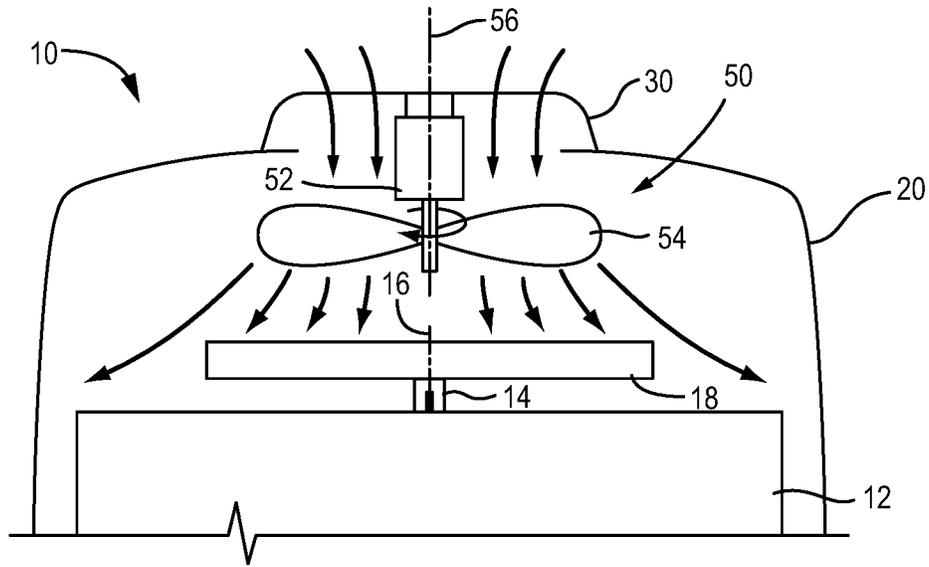


FIG. 7

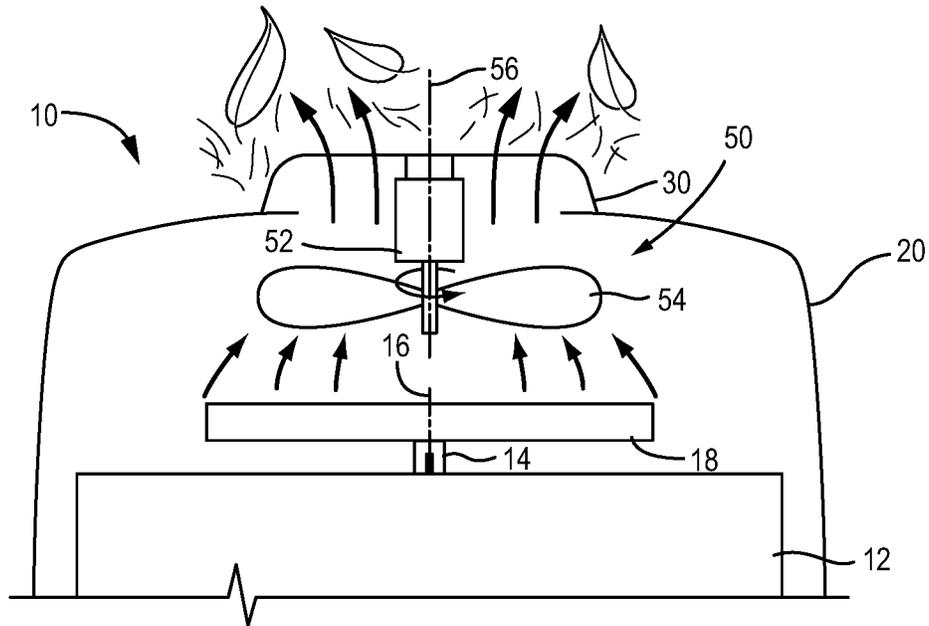


FIG. 8

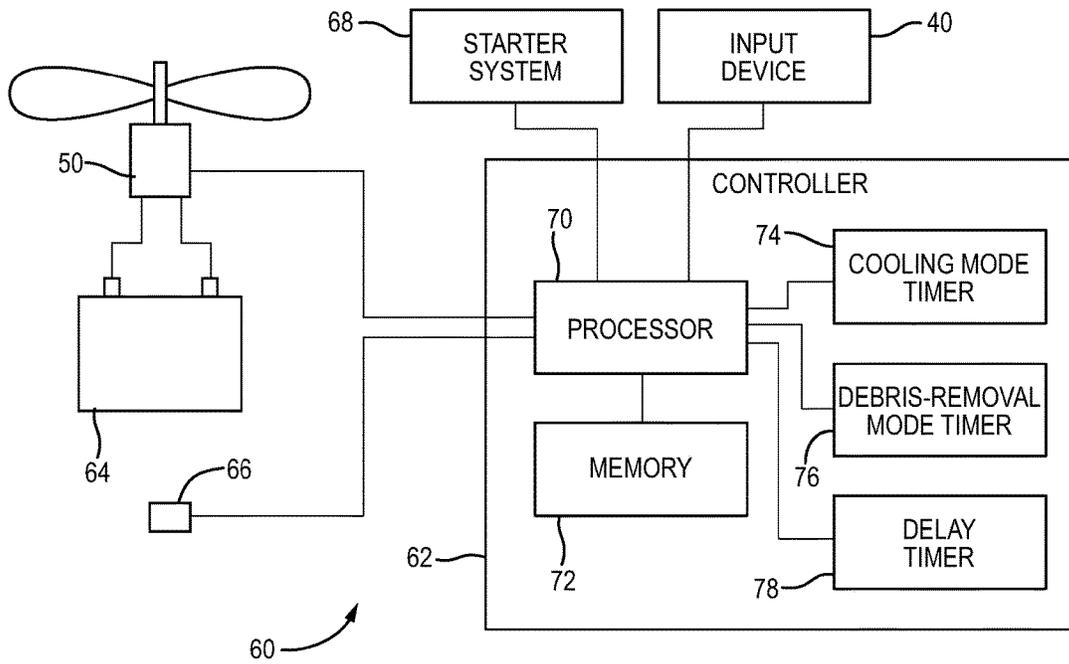


FIG. 9

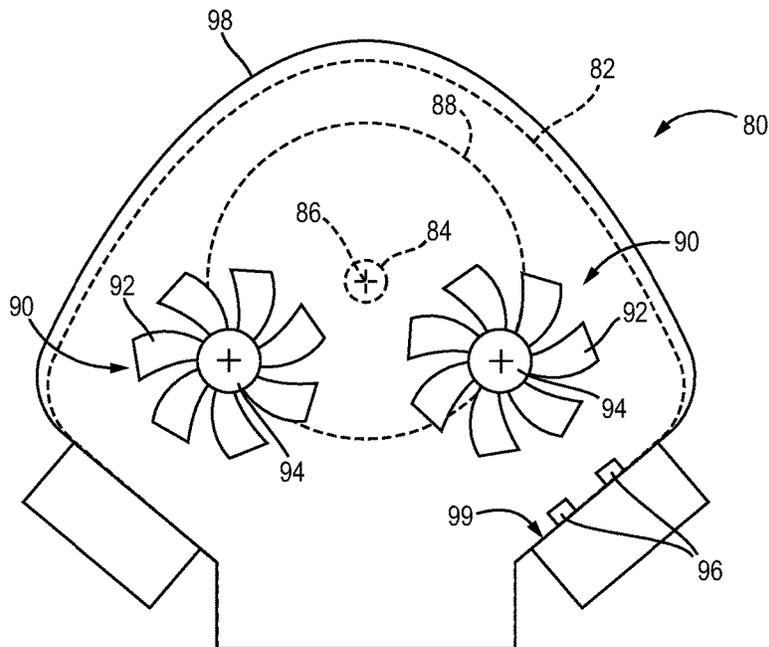


FIG. 10

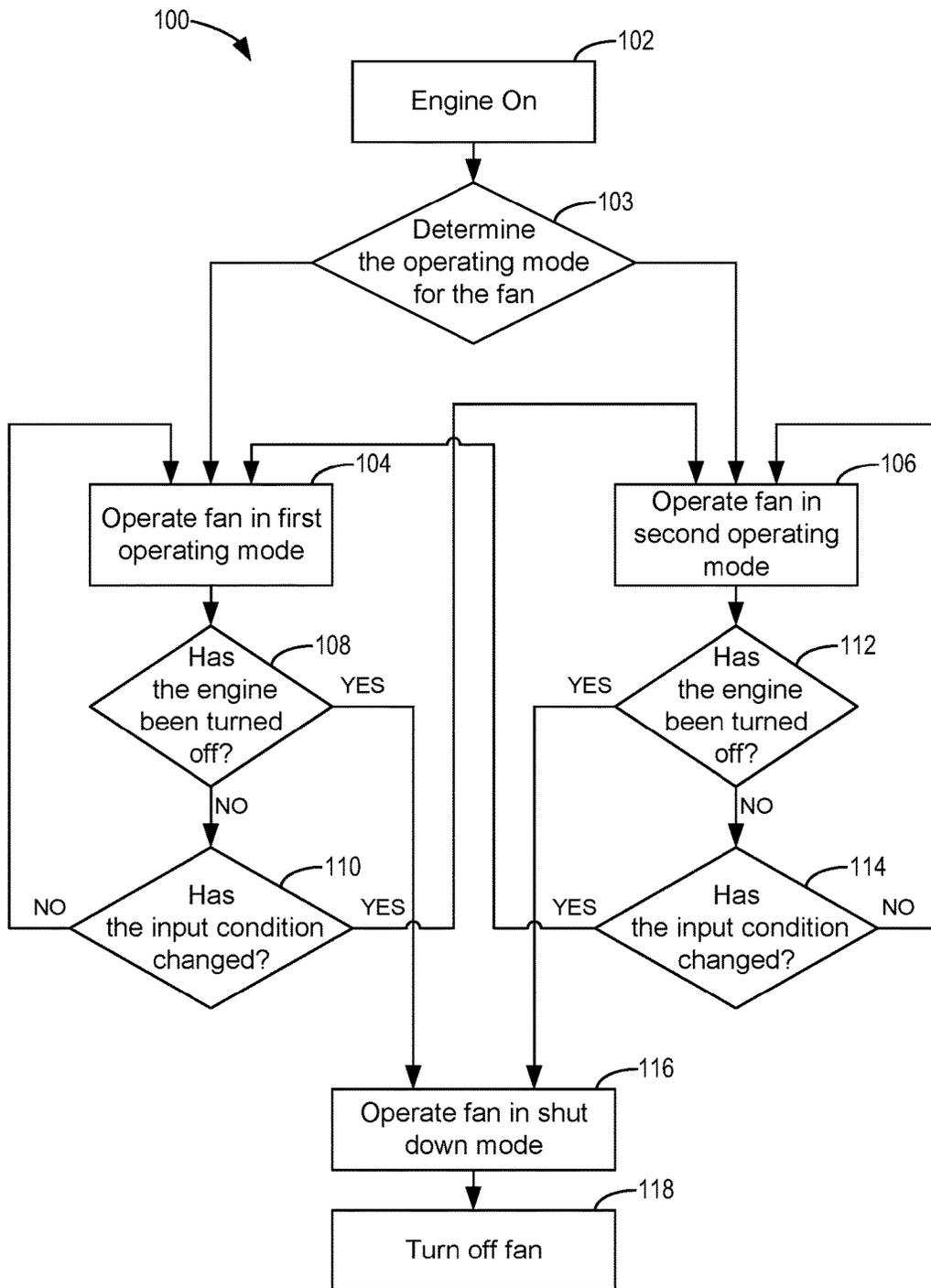


FIG. 11

## COOLING SYSTEM FOR AIR-COOLED ENGINE

### CROSS-REFERENCE TO RELATED PATENT APPLICATIONS

This application is a continuation of U.S. patent application Ser. No. 14/035,469 filed on Sep. 24, 2013, which claims the benefit of U.S. Provisional Application No. 61/777,947, filed Mar. 12, 2013, both of which are incorporated herein by reference in their entireties.

### BACKGROUND

The present invention relates generally to the field of cooling systems for air-cooled internal combustion engines.

Many air-cooled engines include a blower housing and a rotating screen. The rotating screen is positioned over a flywheel and a fan coupled to the crankshaft of engine. As the engine operates, the flywheel, the fan, and the rotating screen rotate with the crankshaft. Under this configuration, cooling air is drawn into the blower housing to cool the engine, while the rotating screen acts to prevent debris from entering blower housing and/or to break or cut any debris entering the blower housing into relatively small pieces. Because of the rotation of rotating screen, debris is not able to quickly build up on the rotating screen and restrict airflow across the engine. However, there may be instances where access to moving parts of the engine, including a rotating screen, must be restricted. Also, the rotating screen may restrict air flow into the blower housing.

Additionally, in many air-cooled engines, the flywheel, the fan, and the rotating screen are coupled to the crankshaft of engine to rotate when the engine is operational. However, due to this coupling, the fan, and the rotating screen are only able to rotate in one direction and are only operational when engine is running. Additionally, because the fan is coupled to the crankshaft, the location and orientation of the fan is limited by the location of the crankshaft.

In engine configurations that do not require the ignition trigger coils to be located adjacent the flywheel (e.g., electronic fuel injection systems), the exact placement of the ignition trigger coils may be varied without adversely affecting the operation of the engine. Some engine manufacturers mount the ignition trigger coils externally on a side of the engine/blower housing. However, this placement opens the ignition trigger coils up to debris and incidental contact.

It would be advantageous to provide a cooling system for an air-cooled internal combustion engine that effectively restricts access to moving parts while still allowing for sufficient airflow and debris management.

### SUMMARY

One embodiment of the invention relates to an air-cooled internal combustion engine that includes a crankshaft, a cylinder, a blower assembly including a blower housing and a fan, and a static cover. The static cover includes a main body that is aligned with the crankshaft, an arm that extends from the main body and is aligned with the cylinder, and a plurality of air intake openings. A first subset of the air intake openings is formed through the main body and a second subset of the air intake openings is formed through the arm, and the static cover is configured to prevent user access to a moving component of the engine. The fan is configured to move air into the blower housing through the air intake openings.

Another embodiment of the invention relates to an air-cooled internal combustion engine that includes a crankshaft, a cylinder, a blower assembly including a blower housing and a fan, and a static cover. The static cover includes a main body that is aligned with the crankshaft, an arm that extends from the main body and aligned with the cylinder, and a plurality of air intake openings. A first subset of the air intake openings is formed through the main body and a second subset of the air intake openings is formed through the arm, and the static cover is configured to prevent user access to a moving component of the engine. The fan is configured to move air into the blower housing through the air intake openings when rotating in a first direction in a cooling mode and to move air out of the blower housing through the air intake openings when rotating in an opposite second direction in a debris-removal mode.

Alternative exemplary embodiments relate to other features and combinations of features as may be generally recited in the claims.

### BRIEF DESCRIPTION OF THE DRAWINGS

The invention will become more fully understood from the following detailed description, taken in conjunction with the accompanying drawings, in which:

FIG. 1 is a perspective view of an air-cooled internal combustion engine, according to an exemplary embodiment.

FIG. 2 is an exploded perspective view of the air-cooled internal combustion engine of FIG. 1, including a centrifugal fan coupled to the flywheel.

FIG. 3 is a cross section view of the air-cooled internal combustion engine along line 3-3 of FIG. 1.

FIG. 4 is a top plan view of the static cover of the air-cooled internal combustion engine of FIG. 1.

FIG. 5 is a side profile view of the static cover of the air-cooled internal combustion engine of FIG. 1.

FIG. 6 is an exploded perspective view of an air-cooled internal combustion engine, according to another exemplary embodiment.

FIG. 7 is a schematic cross-section view of the engine of FIG. 6 with the fan operating in a cooling mode.

FIG. 8 is a schematic cross-section view of the engine of FIG. 6 with the fan operating in a debris-clearing mode.

FIG. 9 is a block diagram of a control system for an electric fan of the engine for an air-cooled engine, according to an exemplary embodiment.

FIG. 10 is a schematic top view of an air-cooled internal combustion including multiple fans, according to an exemplary embodiment.

FIG. 11 is a flow chart illustrating a method of operating a fan for an air-cooled internal combustion engine, according to an exemplary embodiment.

### DETAILED DESCRIPTION

The present invention relates to a cooling system for an air-cooled internal combustion engine. Embodiments of a cooling system in accordance with the present application include a static screen used to shield moving parts of the engine while minimizing airflow restriction and/or one or more electric cooling fans which may be operated in two directions—forward to draw cooling air into and around the engine, and reverse to blow debris away from the air intake vents.

Before turning to the figures, which illustrate the exemplary embodiments in detail, it should be understood that the application is not limited to the details or methodology set

forth in the description or illustrated in the figures. It should also be understood that the terminology is for the purpose of description only and should not be regarded as limiting.

Referring to FIGS. 1-3, an internal combustion engine is shown according to an exemplary embodiment as a small, gasoline-powered, four-stroke cycle engine **10**. The engine **10** includes pistons moveable within cylinders **13** formed in an engine block **12**. The reciprocating motion of the pistons rotates a crankshaft **14** about an axis **16** (see FIG. 3). A flywheel **18** is coupled to the crankshaft **14** and is positioned near the top of the engine **10**, above the engine block **12**. According to an exemplary embodiment, the engine **10** includes two cylinders **13** arranged in a V-twin configuration. However a broad range of engines and other fluid holding components may benefit from the teachings disclosed herein. For example, in some contemplated embodiments, the engine may include a single cylinder or three, or more cylinders in any of a number of different configurations (e.g., inline, horizontally opposed, etc.), or may have a two-stroke cycle. In some embodiments, the engine **10** is vertically shafted (as shown in FIG. 1), while in other embodiments, the engine may be horizontally shafted. The engine **10** may be configured to power a broad range of equipment, including walk behind lawn mowers, zero-turn radius mowers, lawn tractors, pressure washers, electric generators, snow throwers, and other outdoor power equipment.

The engine **10** further includes a blower assembly **19** configured to direct air to the engine block **12** to cool the engine **10** by removing waste heat from the engine block **12**. As shown in FIG. 3, the blower assembly **19** includes a blower housing **20** (e.g., engine cover, engine shroud, etc.) coupled to the top of the engine **10**. The blower housing **20** includes a main body or central portion **22** forming an opening **24** through which the air passes to the engine block **12**. According to an exemplary embodiment, the blower housing **20** is configured for use with the engine **10** having a V-twin arrangement and may be shaped to generally conform with the shape of the engine block **12**. The central portion **22** is aligned with the crankshaft **14**. The blower housing **20** may therefore further include two angled arms **25** extending outward from the central portion **22** that are generally aligned with the cylinders of the engine **10**. Alternatively, for a single cylinder engine, the blower housing **20** includes a single arm **25**.

The blower assembly **19** may be an active system with components that draw air in through a static cover **30** and the blower housing **20** to cool the engine. According to one exemplary embodiment illustrated in FIG. 2, the blower assembly **19** includes a blower fan, shown as a centrifugal fan **26**, and an optional rotating screen (shown as blade-style rotating screen **29**), both of which are coupled to the flywheel **18**. The centrifugal fan **26** rotates about the same axis **16** as the flywheel **18**. The centrifugal fan **26** includes a multitude of fan blades **27** configured to discharge a cooling airflow through an airspace defined between the engine block **12** and the blower housing **20**. The fan blades **27** each include an inner or leading edge defining a central inlet through which axially directed air is drawn. The cooling airflow is discharged from the centrifugal fan **26** in a radially outward direction past the trailing edges of the respective fan blades, and into the airspace between the engine block **12** and the blower housing **20**.

The rotating screen **29** includes a central hub and blades extending outward from the hub to an outer band. The hub, the blades, and the outer band may all be interconnected and integrally formed as a single unitary piece by a suitable

process, such as injection molding or casting. In an exemplary embodiment, the rotating screen **29** included between 4 and 16 blades. According to the embodiment illustrated in FIG. 2, the rotating screen **29** includes 12 blades.

The blades each include a root adjacent the hub and a tip spaced outwardly from the root. In one embodiment, the blades may extend outward in a radial direction (i.e. without skew). In another embodiment, the blades may include a forward or backward skew or the blades may intersect the hub in a substantially tangential manner. The radial distance between the rotational axis of the rotating screen and the tips of the respective blades is defined as the maximum blade radius of the rotating screen, while the radial distance between the root and the tip of each blade is defined as the blade span. Several characteristics of the blade may vary over the span. The blades further include a leading edge between the root and the tip and a trailing edge between the root and the tip relative to the direction of rotation of the rotating screen (e.g., a clockwise rotational direction).

The blade-style rotating screen **29** is further described in commonly-owned U.S. patent application Ser. No. 13/592, 803 filed on Aug. 23, 2012, which is incorporated herein by reference in its entirety.

The engine **10** still further includes the static cover **30** (e.g., stationary screen, grill, non-rotating screen, etc.) that is coupled to the blower housing **20**. The static cover **30** includes multiple intake openings **32** that allow air to pass through the static cover **30** but restrict the intake of debris (e.g., leaves, grass clippings, sticks, etc.). A rotating screen may be coupled to the crankshaft and provided below the static cover **30**. The static cover **30** prevents access to moving components of the engine **10** (e.g., the crankshaft **14**, the flywheel **18**, the rotating screen, fans, etc.).

According to an exemplary embodiment, and shown in more detail in FIGS. 4-5, the static cover **30** is formed in a "V" shape and includes a roughly cylindrically shaped central portion **34** (e.g., main body) with a top **35** and sidewalls **36**. The static cover **30** further includes a pair of arms **38** extending outward from the sidewalls **36** and a flange **39** extending between the arms **38** and the central portion **34**. The static cover **30** may be formed as a single body (i.e., as a unitary component) or may include multiple separate portions. As shown in FIG. 2, the static cover **30** is fastened to an annular flange **23** on the blower housing **20** surrounding the opening **24** and to the arms **25** of the blower housing **20** by conventional fasteners (e.g., screws); however, other suitable fastening means may be employed. For example, the static cover **30** may be coupled to the blower housing **20** with a snap fit using integrally formed fastening features. The static cover **30** is configured to be removable from the blower housing **20**. When the static cover **30** is removed from the blower housing **20**, a user can access various portions of the engine **10** such as the fan, flywheel, the rotating screen, etc. Access doors may be provided on the blower housing **20** to allow the user even greater access to portions of engine **10** underneath the blower housing **20** without necessitating the removal of the blower housing **20**.

The air intake openings **32** of the static cover **30** are not only positioned above the fan/flywheel region of the engine **10**, but also over the respective cylinders of the engine **10**. The intake openings **32** may be formed on any surface of the static cover, including the top **35**, the sidewalls **36**, or the arms **38**. This additional coverage over the respective cylinders of the engine **10** allows for the static cover **30** to include a greater number of air intake openings **32**, which allows for more potential air flow into the blower housing **20**. While air intake openings **32** are shown as slots being

5

formed in a grill-like manner, the static cover **30** is not limited to such a formation. According to other exemplary embodiments, the air intake openings **32** may be slots arranged in other patterns or may be another shape that allow for a sufficient airflow into the blower housing **20** while preventing access to the moving parts and limiting the intake of debris (e.g., perforations, holes, openings, apertures, mesh, a screen, etc.).

In the embodiment illustrated in FIG. 1-3, rotation of the centrifugal fan **26** with the flywheel **18** and the crankshaft **14** about the axis **16** draws air in through the air intake openings **32** in the static cover **30** and into the blower housing **20**. The air is discharged from the centrifugal fan **26** into the airspace between the engine block **12** and the blower housing **20**. The centrifugal fan **26** is coupled to the flywheel **18** and therefore operates concurrently with operation of the engine **10**. In some embodiments, air discharged from the centrifugal fan **26** may be directed to other engine locations, including toward an air cleaner and/or toward an engine air intake. For example, air may be directed toward a cyclonic air cleaner assembly as described in commonly-owned U.S. Pat. No. 8,241,378, which is incorporated herein by reference in its entirety. The static cover **30** does not rotate and allows air to pass through the intake openings **32** while preventing an operator from touching or otherwise contacting rotating components (e.g., the centrifugal fan **26** or the flywheel **18**) and restricting the intake of debris into the airspace between the engine block **12** and the blower housing **20**. Including air intake openings **32** on the top **35**, the sidewalls **36** and the arms **38** of the static cover **30** increases the potential intake area and increases the likelihood that air will be able to be drawn into the blower housing **20** by the centrifugal fan **26** in the event that there is partial blockage of some of the air intake openings **32** by debris.

Mesh-style rotating screens having relatively small openings in the screen (e.g., multiple openings each about 0.14 to about 0.23 inches in diameter providing total open areas of about 25 to 32 square inches) substantially restrict air flow through the blower assembly. When such a mesh-style rotating screen is used in combination with a guard or cover formed as a wire cage with large openings (i.e., larger than the air intake openings **32**), the mesh-style rotating screen is the restriction point for the blower assembly. These large openings may not completely prevent user access to the moving parts of the engine.

The static cover **30** provides significantly more open area than the mesh-style rotating screens described above while preventing user access to the moving parts of the engine. In one embodiment, the static cover **30** provides 42.58 square inches of open area. This increase in open area is achieved by providing air intake openings **32** in the sidewalls **36** and the arms **38** of the static cover. For example, the static cover **30** provides about 56% more open area than a similar static cover in which air intake openings are only formed in the top of the central portion. This similar static cover may be known as "top hat" cover. Providing air intake openings **32** in the arms **38** also helps to minimize the overall height of the static cover **30** by providing air intake openings **32** in locations other than the central portion **34**. A similar static cover with air intake openings only in the central portion would likely need to have a relatively large overall height in order to provide the necessary airflow through the blower assembly.

The combination of the blade-style rotating screen **29** and the static cover **30** reduces the restriction of air flow through the blower assembly **19** as compared to previous cooling systems for air-cooled engines. The restriction point for this

6

combination is the static cover **30**, which has greater open area than mesh-style rotating screens (the restriction point in other blower assembly), thereby restricting air flow through the blower assembly **19** less than a mesh-style rotating screen. The blade-style rotating screen **29** has little to no impact on the overall restriction of air-flow through the blower assembly **19** when used in combination with the static cover **30**.

Referring now to FIG. 6, the engine **10** is shown including a blower assembly **19** according to another exemplary embodiment. The blower assembly **19** includes a blower fan, shown as the fan **50** provided between the static cover **30** and the engine block **12**. In some embodiments, the blower assembly **19** may further include a rotating screen coupled to the crankshaft **14** as described above with reference to FIG. 2. In one embodiment, the fan **50** is an electric fan that includes an electric motor **52** and a multitude of fan blades **54**. The fan **50** may be mounted to any stationary component of the engine **10**, including, but not limited to, the engine block **12**, the blower housing **20**, or the static cover **30**. The electric motor **52** rotates the fan blades **54** about an axis **56** that is independent of the crankshaft **14**. The fan **50** does not need to be placed directly above the crankshaft **14**, as the rotation of fan blades **54** is not related to the rotation of the crankshaft **14** (i.e., the axis of rotation **56** need not be collinear with the axis of rotation of the crankshaft **14**). According to an exemplary embodiment, the fan **50** is a propeller-type fan that creates a moving column of air parallel to the axis **56**. The fan **50** can be mounted in a position that is tilted or angled out of the horizontal plane to direct the column of air to allow for greater airflow to specific parts of the engine **10**.

The fan **50** may be operated in a cooling mode or in a debris-removal mode. In an exemplary embodiment, the engine **10** may include a user input device **40** configured to allow a user to manually switch the fan **50** between the cooling mode or the debris-removal mode. While the user input device **40** is shown in FIG. 6 as a panel including multiple buttons, in other embodiments the user input device **40** may be any suitable device, such as a switch, dial, touchscreen, etc.

In the cooling mode, the motor **52** is configured rotate the fan blades **54** in a first direction (e.g., clockwise) to move air into the blower assembly **19** through an airspace defined between the engine block **12** and the blower housing **20**. Referring to FIG. 7, rotation of the fan **50** about the axis **56** in the cooling mode draws air in through the air intake openings **32** in the static cover **30** and into the blower housing **20**. The air is discharged downward into the airspace between the engine block **12** and the blower housing **20**. The fan **50** is not coupled to the flywheel **18** and may therefore operate independently of the operation of the engine **10** and is not driven by the flywheel **18**, the crankshaft **14**, or by other components related to the internal combustion process (e.g., camshaft, pistons, etc.). The static cover **30** does not rotate and allows air to pass through the intake openings **32** while preventing an operator from touching or otherwise contacting rotating components such as the fan **50** or the flywheel **18** and restricting the intake of debris into the airspace between the engine block **12** and the blower housing **20**.

Operation of the fan **50** in the cooling mode may cause debris to collect over the air intake openings **32** on the top **35**, the sidewalls **36**, and/or the arms **38** of the static cover **30**. If the engine **10** is run for an extended period of time with the fan **50** in the cooling mode, the blockage of the air intake openings **32** can reduce the potential airflow into the

blower housing **20** and reduce the cooling capabilities of the fan **50**. If the engine **10** is positioned on a vehicle out of sight of the operator (e.g., on a typical zero-turn mower with the engine mounted behind the operator), debris may collect on a static screen without the knowledge of the operator. Referring to FIG. **8**, in the debris-removal mode, the motor **52** is configured to rotate the fan blades **54** in an opposite second direction (e.g., counterclockwise) to move air out of the blower assembly **19**. The air is discharged upward, toward the static cover **30** and outward through the air intake openings **32**, dislodging any debris from the outside surface of the static cover **30**.

The rotational speed of the fan **50** may be altered depending upon the temperature of the engine. For example, a sensed temperature rise in the engine block **12** may cause the fan **50** to rotate at a faster speed to increase the airflow. Additionally, the direction of rotation of the fan **50** may be reversed based on the needs of the system. Such sensing being achieved by various techniques described below (temperature, airflow, time, etc.). For example, if it is sensed that the static cover **30** is plugged with debris obstructing the air intake openings **32**, the fan **50** may be reversed from the engine-cooling mode to the debris-removal mode to direct air up, away from the engine block **12** and towards the static cover **30** to clear the collected debris. The timing of conversion of the fan **50** from the cooling mode to the debris-removal mode may be based on a variety of input, such as engine temperature, time, air flow volume, air flow rate, and/or when the engine is turned off.

Alternatively, the engine **10** may include both a blower fan coupled to the flywheel **18** (e.g., centrifugal fan **26**) and a second fan not coupled to the flywheel **18** or crankshaft **14** (e.g., fan **50**). This allows the blower fan to provide a minimum amount of airflow through the blower assembly **19** when the engine **10** is running and for the second fan to supplement this airflow as needed by operating in the cooling mode or to clear debris from the static screen **30** by operating in the debris-removal mode.

Referring now to FIG. **9**, a control system **60** for the fan **50** is shown according to an exemplary embodiment. The control system **60** is configured to manage the operation of the fan **50** (e.g., speed, direction, on/off state, etc.) to achieve a desired cooling of the engine block **12** while keeping the air intake openings **32** of the static cover **30** generally free of accumulated debris. According to an exemplary embodiment, the control system **60** includes control circuitry **62**, a power supply **64**, and one or more sensors **66** monitoring the engine **10**.

In some embodiments, the control circuitry **62** includes a processor **70** and a memory device **72**. The processor **70** can be implemented as a general purpose processor, an application specific integrated circuit (ASIC), one or more field programmable gate arrays (FPGAs), a group of processing components, or other suitable electronic processing components. The memory device **72** (e.g., memory, memory unit, storage device, etc.) is one or more devices (e.g., RAM, ROM, Flash memory, hard disk storage, etc.) for storing data and/or computer code for completing or facilitating the various processes, layers and modules described in the present application. The memory device **72** may be or include volatile memory or non-volatile memory. The memory device **72** may include database components, object code components, script components, or any other type of information structure for supporting the various activities and information structures described in the present application. According to an exemplary embodiment, the memory device **72** is communicably connected to the processor via

the processing circuit and includes computer code for executing (e.g., by processing circuit and/or processor) one or more processes described herein. In another exemplary embodiment, the control circuitry **62** is implemented as non-programmable circuitry, one or more circuit boards, or one or more linear circuits. "Non-programmable circuitry" consists of analog or digital hard circuitry that does not utilize a microcontroller or software. It is believed that embodiments in which the control circuitry is implemented as non-programmable circuitry including discrete components may be less expensive than embodiments implemented with microcontrollers or using software. Such non-programmable circuitry embodiments do not include a microcontroller. Non-programmable circuitry may include multiple discrete components that implement the various operations described herein.

The power supply **64** provides an on-board power source for the fan **50**. According to an exemplary embodiment, the fan **50** is an electric fan and the power supply **64** is a device capable of providing an electric voltage to the fan **50**, such as a battery (e.g., a lead-acid battery, nickel-cadmium battery, lithium polymer battery, lithium-ion battery, etc.) or an ultracapacitor. The fan **50** may be electrically coupled to the power system of a vehicle in which the engine **10** is installed and the power supply **64** may be device such as an onboard battery or an alternator coupled to the crankshaft that is configured to power other electrical systems (e.g., the engine starter motor, lights, gauges, etc.). In another embodiment, the power supply **64** may be a dedicated device providing power only to the fan **50** and the control system **60**. According to another exemplary embodiment, the fan **50** may not be an electric fan and the power supply may store or provide power in another form, such as mechanically or via a hydraulic system.

The sensor **66** monitors the engine **10** such that the fan **50** may be utilized to maintain the engine **10** at a desired operating temperature. In one exemplary embodiment, the sensor **66** may be configured to sense the temperature of a portion of the engine **10** and may be a temperature sensor, such as a conventional oil temperature sensor, cylinder head temperature sensor, bi-metallic temperature sensor used for choking/governing, or dedicated temperature sensor(s) used solely for fan operation. In another exemplary embodiment, the sensor **66** may be configured to sense the airflow through the static screen and/or downstream of the fan **50** and may be an air flow sensor such as a vane meter sensor, hot wire sensor, membrane sensor, or may be a pressure sensor (e.g., a differential pressure sensor that measures the difference in pressure across the static cover **30**). The control system **60** may include multiple sensors **66** positioned in various portions of the engine **10**. The control circuitry **62** may further be configured to monitor other engine systems, such as the state of a starter system **68** for the engine **10**.

According to one exemplary embodiment, the fan **50** is configured to operate in different modes based on a specified timing. For example, the control circuitry **62** may operate the fan **50** in the debris-removal mode for a first time and operate the fan **50** in the cooling mode for a second time. In some embodiments, the first time and the second time are not equal. The first time and the second time may be controlled by inputs from one or more timers (e.g., a cooling mode timer **74**, a debris-removal timer **76**), by one or more sensors (e.g. sensor **66**), or by the user (e.g., via the user input device **40**). For example, the fan **50** may be configured to operate in the debris-removal mode for a period of time when the engine **10** is initially turned on and then periodically for as long as the engine **10** is running (e.g., for 10

seconds every 3 minutes). The control circuitry 62 may include the cooling mode timer 74, the debris-removal mode timer 76, and a delay timer 78. When the engine 10 is started, the control circuitry 62 starts the fan 50 in the debris-removal mode and begins the debris-removal mode timer 76. Once the debris-removal mode timer 76 expires, the control circuitry 62 switches the fan 50 to operate in the cooling mode and starts the cooling mode timer 74. Once the cooling mode timer 74 expires, the control circuitry 62 switches the fan 50 to operate in the debris-removal mode again and starts the debris-removal mode timer 76, beginning the cycle again. When the engine 10 is stopped, the control circuitry 62 may immediately stop the fan 50 or may continue to operate the fan 50 in either the cooling mode or the debris-removal mode for a time period after the engine 10 is stopped.

In another embodiment, the control circuitry 62 normally operates the fan 50 in the cooling mode until a sensor input indicates an elevated temperature above a predetermined threshold, at which time the control circuitry 62 directs the fan 50 to operate in the debris-removal mode. For example, the fan 50 may run in the cooling mode until an elevated temperature in the engine block 12 is detected by the sensor 66 (e.g., due to an obstructed airflow and an insufficient flow of cooling air). The control circuitry 62 then switches the fan 50 to operate in the debris-removal mode. The control circuitry 62 may switch the fan 50 back to the cooling mode after a specified time (e.g., using the debris-removal mode timer 76 as described above). Alternatively, the control circuitry 62 may switch the fan 50 back to the cooling mode when the temperature drops below the previously mentioned threshold or when the temperature drops below a second, predetermined threshold lower than the first predetermined threshold.

In another embodiment, the control circuitry 62 normally operates the fan 50 in the cooling mode until a sensor input indicates an airflow below a predetermined threshold, at which time the control circuitry 62 directs the fan 50 to operate in the debris-removal mode. For example, the fan 50 may run in the cooling mode until a reduced airflow is detected by the sensor 66. The control circuitry 62 then switches the fan 50 to operate in the debris-removal mode. The control circuitry 62 may switch the fan 50 back to the cooling mode after a specified time (e.g., using the debris-removal mode timer 76 as described above) or once the sensor 66 detects that the airflow has exceeded a predetermined threshold indicating that the air intake openings 32 in the static cover 30 are unobstructed. Alternatively, the control circuitry 62 may switch the fan 50 back to the cooling mode when the airflow exceeds a second, predetermined threshold higher than the first predetermined threshold.

In the cooling mode, the control circuitry 62 may alter the behavior of the fan 50 to achieve a desired cooling of the engine block 12. In one embodiment, the fan 50 is a variable speed fan. The control circuitry 62 may adjust the speed of the fan 50 based on input from the sensor 66. For example, if the sensor 66 detects an increased temperature in the engine block 12, the control circuitry 62 may increase the speed of the fan to provide additional cooling air. The control circuitry 62 may switch the fan 50 to the debris-removal mode periodically (e.g., every three minutes) or based on input from the sensor 66 to clear any accumulated debris from the static cover 30. For example, the control circuitry 62 may compare a first sensed engine temperature taken at a lower fan speed to a second sensed engine temperature taken at a higher fan speed a specified time

period after the first sensed temperature. If the second temperature is not lower than the first temperature, it may be determined that the static screen 30 is obstructed. The control circuitry 62 may then switch the fan 50 to operate in the debris-removal mode.

In another embodiment, the fan 50 may only be operated periodically. When the engine 10 is started, the control circuitry 62 starts the fan 50 in the cooling mode and starts the cooling mode timer 74. Once the cooling mode timer 74 expires, the control circuitry 62 turns the fan 50 off and starts the delay timer 78. Once the delay timer 78 expires, the control circuitry 62 turns the fan 50 on again in the cooling mode and starts the cooling mode timer 74, beginning the cycle again. The durations of the cooling mode timer 74 and the delay timer 78 may be static values, may be user configurable, or may be adjusted automatically based on input from the sensor 66. The on/off cycle may be interrupted periodically by the control circuitry 62 to switch the fan 50 to the debris-removal mode. The control circuitry 62 may switch the fan 50 to the debris-removal mode periodically (e.g., every three minutes) or based on input from the sensor 66 to clear any accumulated debris from the static cover 30. Alternatively, the control circuitry 62 may start the fan 50 in the debris-removal mode before intermittently operating the fan 50 in the cooling mode.

Referring to FIG. 10, in another embodiment, an engine 80 includes multiple fans 90 having fan blades 92 driven by electric motors 94. The use of multiple fans 90 allows the fans 90 to be smaller in size than the single fan 50 and allows the fans 90 to be placed at multiple locations on the engine, increasing space for other engine components or allowing for unique engine configurations. Because the fans 90 are not coupled to a flywheel 88 or a crankshaft 84 of the engine 80, the fans 90 may be positioned such that their respective axes of rotation are tilted or angled relative to an axis 86 of the crankshaft 84 and relative to each other. The fans 90 may therefore be independently positioned to maximize the cooling performance over various engine parts. According to an exemplary embodiment, the fans 90 may be positioned to maximize the cooling air provided to the individual cylinders 13 (e.g., positioned above each of the cylinder heads).

The fans 90 may both be controlled by a single control system, similar to the control system 60 described above. The single controller may control the multiple fans 90 independently or in parallel. In another embodiment, each of the fans 90 may be controlled by a separate, independent control system. The control system(s) may alter the direction of rotation of the fans 90 between a cooling mode and a debris-removal mode to provide sufficient cooling to the engine block 82 and to remove any accumulated debris from a static cover covering the fans 90.

According to still other exemplary embodiments, the engine 10 may be equipped with three or more fans. For example, the engine 10 may include three or more fans positioned above three or more cylinder heads to maximize the cooling air provided to the individual cylinders.

In configurations that do not require ignition trigger coils 96 to be located adjacent the flywheel 88 (e.g., electronic fuel injection systems), the exact placement of the ignition trigger coils 96 may be varied without adversely affecting the operation of the engine 80. Referring still to FIG. 10, according to an exemplary embodiment, the ignition trigger coils 96 of the engine 80 may be mounted at a location away from the flywheel 88, such as on an inside surface 99 of a blower housing 98 coupled to the engine 80. The ignition trigger coils 96 may be premounted to the blower housing 98 during the engine assembly process, saving assembly time

and cost. The positioning of the ignition trigger coils 96 inside the blower housing 98 therefore not only protects the trigger coils 96 from debris and incidental contact during use of the engine 80, but also facilitates the assembly of the engine 80.

Referring to FIG. 11, a method of operating a fan for an air-cooled internal combustion engine 100 is illustrated, according to an exemplary embodiment. When the engine (e.g., engine 10, engine 80) is first turned on or started (step 102), the control system determines in what operating mode the fan (e.g., fan 50, fans 90) is to run (step 103). The fan may be run such that it rotates in a first direction in a first operating mode (e.g., in a cooling mode) (step 104) or rotates in a second opposite direction in a second operating mode (e.g., in a debris-removal mode) (step 106). Which of the two operating modes is first used upon starting the engine can be predetermined (i.e., after starting engine always operate in the first mode or always operate in the second mode) or determined based on an input. In some embodiments, the input is provided by one or more sensors (e.g. sensor 66), for example, that the engine temperature is above or below a threshold temperature or that air flow is above or below a threshold air flow. In other embodiments, the input may be provided by a predetermined operating instruction or by a manual user input (e.g., via user input device 40).

With the fan being operated in the first operating mode, the control system monitors the state of the engine (step 108). If the engine is running, the control system monitors various input conditions related to the fan (step 110). The input condition is utilized to determine if the fan should operate in the first operating mode or the second operating mode. In some embodiments, the input condition is provided by a first operating mode timer (e.g., cooling mode timer 74). In some embodiments, the input conditions may be provided by one or more sensors (e.g., temperature sensors, airflow sensors, etc.) or a manual user input or activation. The fan remains in the first operating mode unless the input condition changes. A changing input condition may be, for example, the expiration of the first operating mode timer, a change in temperature beyond a predetermined threshold, a reduction in airflow below a predetermined threshold, a manual user input for changing modes, etc. If the input condition changes, the fan switches from the first operating mode to the second operating mode.

The control system continues to monitor the state of the engine (step 112) with the fan in the second operating mode. The control system also continues to monitor the input conditions related to the fan (step 114). In some embodiments, the input condition is provided by a second operating mode timer (e.g., debris-removal mode timer 76). In some embodiments, the input conditions may be provided by one or more sensors (e.g., temperature sensors, airflow sensors, etc.) or a manual user input or activation. The fan remains in the second operating mode unless the input condition changes. A changing input condition may be, for example, the expiration of the second operating mode timer, a change in temperature beyond a predetermined threshold, an increase in airflow above a predetermined threshold, a manual user input for changing modes, etc. If the input condition changes, the fan switches from the second operating mode to the first operating mode.

The control system continues to monitor the input condition to switch the fan between the first operating mode and the second operating mode until the engine is turned off. If the engine is turned off, the fan may operate in a shutdown mode (step 116) before turning off (step 118). In the shut-

down mode, the fan may continue to operate for a predetermined amount of time in one of the operating modes, for a predetermined amount of time in one of the two operating modes and then for a predetermined amount of time in the other of the two operating modes, or in one of the two operating modes until receiving an input from a sensor (e.g., sensor 66), for example, that the engine temperature is below a threshold temperature indicating that the engine is cooled down or that air flow is above a threshold air flow indicating that debris has been removed from a static cover (e.g., static cover 30). The amount of time in which the fan operates in the shutdown mode may be determined by a first shutdown timer, by a second shutdown timer, by the first operating mode timer or the second operating mode timer. The second shutdown timer may be set for a different amount of time than the first shutdown timer. In other embodiments, the fan may not operate in either the first or the second operating modes in shutdown mode and may stop immediately when the engine is turned off.

The construction and arrangement of the apparatus, systems and methods as shown in the various exemplary embodiments are illustrative only. Although only a few embodiments have been described in detail in this disclosure, many modifications are possible (e.g., variations in sizes, dimensions, structures, shapes and proportions of the various elements, values of parameters, mounting arrangements, use of materials, colors, orientations, etc.). For example, some elements shown as integrally formed may be constructed from multiple parts or elements, the position of elements may be reversed or otherwise varied and the nature or number of discrete elements or positions may be altered or varied. Accordingly, all such modifications are intended to be included within the scope of the present disclosure. The order or sequence of any process or method steps may be varied or re-sequenced according to alternative embodiments. Other substitutions, modifications, changes, and omissions may be made in the design, operating conditions and arrangement of the exemplary embodiments without departing from the scope of the present disclosure.

The present disclosure contemplates methods, systems and program products on any machine-readable media for accomplishing various operations. The embodiments of the present disclosure may be implemented using existing computer processors, or by a special purpose computer processor for an appropriate system, incorporated for this or another purpose, or by a hardwired system. Embodiments within the scope of the present disclosure include program products comprising machine-readable media for carrying or having machine-executable instructions or data structures stored thereon. Such machine-readable media can be any available media that can be accessed by a general purpose or special purpose computer or other machine with a processor. By way of example, such machine-readable media can comprise RAM, ROM, EPROM, EEPROM, CD-ROM or other optical disk storage, magnetic disk storage or other magnetic storage devices, or any other medium which can be used to carry or store desired program code in the form of machine-executable instructions or data structures and which can be accessed by a general purpose or special purpose computer or other machine with a processor. When information is transferred or provided over a network or another communications connection (either hardwired, wireless, or a combination of hardwired or wireless) to a machine, the machine properly views the connection as a machine-readable medium. Thus, any such connection is properly termed a machine-readable medium. Combinations of the above are also included within the scope of machine-readable media.

## 13

Machine-executable instructions include, for example, instructions and data which cause a general purpose computer, special purpose computer, or special purpose processing machines to perform a certain function or group of functions.

Although the figures may show or the description may provide a specific order of method steps, the order of the steps may differ from what is depicted. Also two or more steps may be performed concurrently or with partial concurrence. Such variation will depend on various factors, including software and hardware systems chosen and on designer choice. All such variations are within the scope of the disclosure. Likewise, software implementations could be accomplished with standard programming techniques with rule based logic and other logic to accomplish the various connection steps, processing steps, comparison steps and decision steps.

What is claimed is:

1. An air-cooled internal combustion engine, comprising:
  - a crankshaft;
  - a first cylinder and a second cylinder;
  - a blower assembly including a blower housing and a fan; and
  - a static cover including
    - a main body aligned with the crankshaft,
    - a first arm extending from the main body and aligned with the first cylinder, and
    - a plurality of air intake openings, wherein a first subset of the air intake openings is formed through the main body and a second subset of the air intake openings is formed through the arm, and
    - wherein the static cover inhibits user access to a moving component of the engine;
  - wherein the fan is configured to move air into the blower housing through the air intake openings and substantially all the air entering the blower housing flows through the static cover;
  - wherein the static cover includes a second arm extending from the main body and aligned with the second cylinder and wherein a third subset of plurality of air intake openings is formed through the second arm.
2. The air-cooled internal combustion engine of claim 1, wherein the first subset of the air intake openings is located above the crankshaft and the second subset of the air intake openings is located above the first cylinder.
3. The air-cooled internal combustion engine of claim 1, wherein the main body includes a top surface and a side surface extending from the top surface; and
  - wherein the first subset of air intake openings is formed through the top surface and the side surface.
4. The air-cooled internal combustion engine of claim 1, further comprising a starter system located remote from the static cover.
5. The air-cooled internal combustion engine of claim 1, wherein the first arm includes an arm top surface and the second subset of air intake openings is formed through the arm top surface.
6. The air-cooled internal combustion engine of claim 1, wherein the main body includes a top surface and a side surface extending from the top surface; and
  - wherein the first subset of air intake openings is formed through the top surface and the side surface.
7. The air-cooled internal combustion engine of claim 6, wherein the first arm includes a first arm top surface and the second subset of air intake openings is formed through the first arm top surface; and

## 14

wherein the second arm includes a second arm top surface and the third subset of air intake openings is formed through the second arm top surface.

8. The air-cooled internal combustion engine of claim 1, wherein the first arm includes a first arm top surface and the second subset of air intake openings is formed through the first arm top surface; and
  - wherein the second arm includes a second arm top surface and the third subset of air intake openings is formed through the second arm top surface.
9. The air-cooled internal combustion engine of claim 1, wherein the static cover is formed as a unitary component.
10. The air-cooled internal combustion engine of claim 1, wherein the fan is configured to move air into the blower housing through the air intake openings when rotating in a first direction in a cooling mode and to move air out of the blower housing through the air intake openings when rotating in an opposite second direction in a debris-removal mode.
11. The air-cooled internal combustion engine of claim 1, wherein the fan is coupled to the crankshaft for rotation with the crankshaft and configured to move air into the blower housing through the air intake openings when rotating.
12. The air-cooled internal combustion engine of claim 1, further comprising: a rotating screen coupled to and driven by the crankshaft.
13. An air-cooled internal combustion engine, comprising:
  - a crankshaft;
  - a cylinder;
  - a blower assembly including a blower housing and a first fan;
  - a static cover including
    - a main body aligned with the crankshaft,
    - an arm extending from the main body and aligned with the cylinder, and
    - a plurality of air intake openings, wherein a first subset of the air intake openings is formed through the main body and a second subset of the air intake openings is formed through the arm, and
    - wherein the static cover inhibits user access to a moving component of the engine; wherein the first fan is configured to move air into the blower housing through the air intake openings and substantially all the air entering the blower housing flows through the static cover; and
  - a second fan configured to move air into the blower housing through the air intake openings when rotating in a first direction in a cooling mode and to move air out of the blower housing through the air intake openings when rotating in an opposite second direction in a debris-removal mode.
14. An air-cooled internal combustion engine, comprising:
  - a crankshaft;
  - a cylinder;
  - a blower assembly including a blower housing and a fan;
  - a static cover including
    - a main body aligned with the crankshaft,
    - an arm extending from the main body and aligned with the cylinder, and
    - a plurality of air intake openings, wherein a first subset of the air intake openings is formed through the main body and a second subset of the air intake openings is formed through the arm, and

**15**

wherein the static cover is configured to prevent user access to a moving component of the engine; a sensor structured to measure an operating condition of the air-cooled internal combustion engine; and a control circuit structured to receiving information from the sensor and operate in a cooling mode and a debris-removal mode based on the receive information, wherein the cooling mode moves air into the blower housing through the air intake openings when rotating in a first direction and the debris-removal mode moves air out of the blower housing through the air intake openings when rotating in an opposite second direction.

**15.** The air-cooled internal combustion engine of claim **14**, wherein the first subset of the air intake openings is located above the crankshaft and the second subset of the air intake openings is located above the cylinder.

**16.** The air-cooled internal combustion engine of claim **14**, wherein the main body includes a top surface and a side surface extending from the top surface; and

**16**

wherein the first subset of air intake openings is formed through the top surface and the side surface.

**17.** The air-cooled internal combustion engine of claim **14**, wherein the sensor is structured to measure one of elapsed time, engine load, engine speed, or engine temperature.

**18.** The air-cooled internal combustion engine of claim **14**, wherein the arm includes an arm top surface and the second subset of air intake openings is formed through the arm top surface.

**19.** The air-cooled internal combustion engine of claim **14**, further comprising a second cylinder, wherein the static cover includes a second arm extending from the main body and aligned with the second cylinder and wherein a third subset of the plurality of air intake openings is formed through the second arm.

\* \* \* \* \*