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ABSTRACT

A brake pipe sensing unit may be used with a universal pneumatic
brake control unit of a rail vehicle. The brake pipe sensing unit
5 includes a control reservoir, an auxiliary reservoir, a three state
valve, a quick service valve, a brake pipe transducer and a
transmission device. The control reservoir is used as a volume in
which gas under pressure may stored and from which gas under
pressure may be supplied. The auxiliary reservoir communicates
10 with a supply port of the universal unit and with a first and a
second route of the brake pipe sensing unit. The three state valve
has a first pilot port in communication with a brake pipe of the
rail vehicle, a second pilot port in communication with the control
reservoir, and first and second exhaust ports both of which connect
15 to atmosphere. The three state valve operates in an apply
position, a release position and a lap position depending on the
pressure bearing on the first and second pilot ports. The quick
service valve connects within the first route and has a service
pilot port in communication with both the auxiliary reservoir and
20 the supply port. The quick service valve closes the first route
when pressure at the service pilot port exceeds a preselected
level. The brake pipe transducer converts pressure within the
brake pipe to an electrical signal. The transmission device
communicates the electrical signal received from the brake pipe
25 transducer to a brake control bus on the rail vehicle.

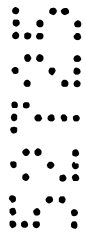
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Invention Title: **BRAKE PIPE SENSING UNIT**

The following statement is a full description of this invention, including the best method of performing it known to us

BRAKE PIPE SENSING UNIT

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FIELD OF THE INVENTION

The present invention generally relates to a brake control unit for controlling pressure supplied to the brake cylinders of one or more trucks of a rail vehicle. More particularly, the present invention relates to a brake pipe sensing unit which can be used as a pneumatic backup to the brake control unit that otherwise controls the braking on each of the trucks.

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BACKGROUND OF THE INVENTION

It is well known in the brake control art that modern train brake control systems typically use a central controller unit to control the brakes of the rail vehicles that comprise the train. A train operator located in the lead locomotive manipulates the brake handles or like devices of the train to apply and release the brakes of the trucks on each rail vehicle as desired. The inputs from the brake handles are typically processed by a cab control unit and passed to the central controller unit. In response to

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these and other inputs, the central controller unit issues a brake command signal along a trainline to each of the rail vehicles in the form of either a pneumatic signal or an electrical signal or even both. Brake equipment on each of the rail vehicles applies or releases the brakes according to the dictates of the particular
5 brake command signal received.

Depending on the type of train being considered, the brake equipment on each rail vehicle may include either exclusively pneumatic equipment or a combination of electrical and pneumatic
10 (i.e., electropneumatic) equipment. In those trains featuring only pneumatic brake equipment on the rail vehicles, the central controller unit sends the brake command signal along a pneumatic trainline (i.e., pipe) to each of the rail vehicles. The pneumatic brake equipment on each rail vehicle responds pneumatically to
15 apply or release the brakes according to the dictates of the pneumatic brake command signal. In those trains featuring electropneumatic brake equipment on the rail vehicles, each rail vehicle typically includes a master electronic unit (MEU) whose construction and operation are generally well known in the brake control art. The central controller unit sends the brake command
20 signal to each of the rail vehicles along either a pneumatic trainline or an electrical trainline.

The MEU on each rail vehicle receives the brake command signal and various other signals in response to which it directly
25 controls the electropneumatic equipment according to principles well known in the brake control art. Specifically, the MEU

directly generates the electrical signals which open or close the various valves which supply pressure to or vent pressure from the brake cylinders. The brakes on each rail vehicle apply and release accordingly.

5 The MEU, for example, reads the rate request signal, the power/brake signal and the emergency brake signal received from the brake control system. The rate request signal is basically the brake command signal and represents the desired rate at which the brakes of the train are to be applied. Depending on the type of
10 brake control system, the rate request signal may be representative not only of the brake command signal but also the rate at which to propel the rail vehicle during propulsion. As is well known in the brake control art, the power/brake signal is used in conjunction with the rate request signal to distinguish whether the train is in
15 a braking mode or a propulsion mode.

The MEU uses the rate request, the power/brake and the emergency brake signals to determine the rate at which the entire train is to be braked, i.e., train braking rate request in miles per hour per second (mphps). The MEU typically converts this train
20 braking rate request into the rates at which the brakes on its rail vehicle are to be applied, i.e., the railcar braking rate request. The railcar braking rate request (mphps) represents the amount of braking force that should be applied by the brakes of a given rail vehicle.

25 The MEU may also receive several other signals such as a speed signal, an air spring pressure signal, a dynamic brake

feedback signal, a snow brake request signal and a wheel slip control signal. The MEU may use the speed signal to perform speed tapering and jerk limiting. Speed tapering can be employed at speeds above a predetermined level to lower the rate of braking on each rail vehicle. This would decrease the likelihood of wheel slippage and reduce thermal loading on the brakes. This yields a modified railcar braking request. As sudden changes in the speed and motion of a rail vehicle may be undesirable especially in passenger transit applications, the modified truck braking rate request can be adjusted to substantially reduce the amount of jerking that would otherwise be encountered with quick acceleration or braking. The rate at which the railcar brakes are applied or the rate at which the rail vehicle is accelerated may be limited to a preset jerk limit expressed in miles per hour per second per second (mphpsps). This yields a jerk limited railcar braking request.

The air spring pressure signal may be used to estimate the weight of the rail vehicle and modify the jerk limited railcar braking request. This compensates for the weight of the load borne by the rail vehicle during brake applications. Based on rail vehicle weight and the jerk limited railcar braking request, the MEU calculates the required friction tractive effort to be applied by the brakes of each railcar. The wheel slip control signal can be used to reduce the friction tractive effort in response to wheel slippage. The friction tractive effort may be further reduced by

the amount of dynamic braking effort provided by the propulsion units.

5 The MEU may also take into account the snow brake request. This signal represents the small amount of braking force that may be applied to the railcar wheels to prevent build up of ice and/or snow. The MEU may further reduce the friction tractive effort to account for the snow braking effort already applied to the wheels. This yields the final braking effort to be applied to the wheels of each rail vehicle. The final braking effort
10 represents the amount of pressure that ought to be supplied to the brake cylinders of the rail vehicle.

The MEU converts the final friction braking effort into the electrical signals that are used to directly control the electropneumatic equipment according to the aforementioned well
15 known principles. Specifically, it is through these electrical signals that the MEU opens or closes the valves which supply pressure to or vent pressure from the brake cylinders thereby applying or releasing the brakes of all trucks on the rail vehicle.

20 The prior art brake control systems discussed in the foregoing paragraphs have certain disadvantages when compared to the invention set forth below and related inventions set forth in related applications. The prior art systems typically provide service braking, emergency braking and wheel slip and spin control functions from separate components. This requires a lot of space.
25 Moreover, each component in these prior art systems is typically physically separated from the electronics that controls it. Large

amounts of wiring are therefore required to interconnect each component and its controlling electronics. Each rail vehicle, of course, bears the weight of this wiring.

5 It would therefore be highly desirable to consolidate the service braking, emergency braking and wheel slip and spin control components and their associated electronics within a single, and comparatively small, package. The amount of wiring that would be required to interconnect the various electropneumatic components and their associated electronics in each package would be vastly
10 reduced as compared to the prior art systems. This would reduce the weight that each rail vehicle would be required to bear. The single enclosure for each of the components and their associated electronics would also afford better protection from adverse environmental conditions.

15 It would be also be highly desirable to design such a package that either by itself or in conjunction with another novel device can be used within and made a part of a variety of train brake control systems such as those used to control passenger trains, subway trains and trolleys. The design of the package
20 would preferably allow it to be substituted not only for the components used to implement service braking and emergency braking in such prior art systems but also, if desired, for the components used to implement wheel slip and spin control.

25 It would be also be highly desirable to control the brakes at the truck level. In those prior art systems featuring only pneumatic brake equipment on each rail vehicle, the brakes are

controlled at the rail vehicle level. Similarly, in those prior art brake control systems featuring electropneumatic brake equipment on each rail vehicle, the MEU is used to control the brakes at the rail vehicle level. By controlling the brakes at each truck individually, this would allow for better overall control of the brakes given the inevitable variations in load weight, motion, wheel slippage and other factors affecting each rail vehicle.

It should be noted that the foregoing background information is provided to assist the reader in understanding the present invention. Accordingly, any terms of art used herein are not intended to be limited to any particular narrow interpretation unless specifically stated otherwise in this document.

SUMMARY OF THE INVENTION

The present invention therefore provides a brake pipe sensing unit for use with a universal pneumatic brake control unit of a rail vehicle. In a presently preferred embodiment, the brake pipe sensing unit includes a control reservoir, an auxiliary reservoir, a three state valve, a quick service valve, a brake pipe transducer and a transmission means. The control reservoir is a volume within which gas under pressure may stored and from which gas under pressure may be supplied. The auxiliary reservoir communicates with a supply port of the universal unit and with a first and a second route of the brake pipe sensing unit. The three state valve has a first pilot port in communication with a brake pipe of the rail vehicle, a second pilot port in communication with

the control reservoir, and first and second exhaust ports both of which connect to atmosphere. The three state valve operates in an apply position when pressure at the first pilot port drops relative to pressure at the second pilot port. This allows gas to flow from the brake pipe through the three state valve and the first route both to the auxiliary reservoir and to the supply port. Gas also flows from the control reservoir through the three state valve and the second route both to the auxiliary reservoir and to the supply port. The three state valve operates in a release position when pressure at the first pilot port increases relative to pressure at the second pilot port. This allows gas to flow from the brake pipe through the three state valve to the control reservoir. Gas also flows from both the auxiliary reservoir and the supply port through the first and second routes through the three state valve to atmosphere via the first and second exhaust ports. The three state valve operates in a lap position when pressure at the first pilot port equalizes relative to pressure at the second pilot port. This allows gas to flow from the brake pipe to the control reservoir. Gas also flows from both the auxiliary reservoir and the supply port through the first route and the three state valve to atmosphere via the second exhaust port. The quick service valve connects within the first route and has a service pilot port in communication with both the auxiliary reservoir and the supply port. The quick service valve closes the first route when pressure at its service pilot port exceeds a preselected level. The brake pipe transducer converts the pressure within the brake pipe to an

electrical brake command signal. The transmission means communicates the brake command signal received from the brake pipe transducer to a brake control bus on the rail vehicle.

OBJECTS OF THE INVENTION

5 It is, therefore, a primary object of the present invention to provide a brake pipe sensing unit that serves as a pneumatic backup to a universal pneumatic brake control unit so as to assure application of the brakes of one or more trucks of a rail vehicle.

10 Another object of the present invention is to provide a brake pipe sensing unit that serves as a source of pressure from which to supply a universal pneumatic brake control unit.

15 Yet another object of the present invention is to provide a brake pipe sensing unit that converts pressure within a pneumatic trainline to an electrical signal and conveys the electrical signal to other parts of a brake control system.

20 In addition to the objects and advantages listed above, various other objects and advantages of the present invention will become more readily apparent to persons skilled in the relevant art from a reading of the detailed description section of this document. Such other objects and advantages will become particularly apparent when the detailed description is considered along with the attached drawings and with the appended claims.

BRIEF DESCRIPTION OF THE DRAWINGS

Figure 1 is a partial schematic view of a brake control system in which the present invention has been incorporated along with a universal brake control unit.

5 Figure 2 is a schematic view of the universal brake control unit with which the present invention may cooperate.

Figure 3 is a schematic view of the universal brake control unit with which the present invention may cooperate.

10 Figure 4 is a block diagram that illustrates a brake control process.

Figure 5 is a schematic view of a brake assurance module of the universal brake control unit shown in Figures 2 and 3.

DETAILED DESCRIPTION OF THE INVENTION

15 Before describing the present invention in detail, the reader is advised that, for the sake of clarity and understanding, identical components having identical functions in each of the accompanying drawings have been marked with the same reference numerals in each of the Figures provided in this document.

20 The present invention can be used within and made a part of a variety of train brake control systems. The present invention, for example, may be incorporated into pneumatic brake control systems such as the WABCO 26-C Brake Control System. Depending on the particular brake control system in which the present invention is used, certain modifications that should be
25 apparent to persons skilled in the brake control art may be necessary. Any of the brake control systems featuring the present

invention will more reliably control the brakes of the train as compared to such brake control systems lacking the present invention.

5 Figure 1, for example, illustrates the essential details of a presently preferred embodiment of the present invention. This figure depicts the present invention within the WABCO 26-C Brake Control System, generally designated 1, to control the brakes of a train. The brake control system includes a universal pneumatic
10 brake control unit 100 through which to control the brakes on one or more trucks of a given rail vehicle. For reasons that will become more apparent from a reading of the ensuing paragraphs, one universal unit is preferably used for each truck.

Referring now to Figures 2 and 3, the universal unit basically comprises an electronics section 200, an electropneumatic
15 section 400 and a pneumatic section 600. The electronic section 200 includes a communication interface means 210, a local control means 230, a speed sensor interface means 240, a solenoid driver means 250, a transducer interface means 260 and a brake assurance
20 interface means 300. The electropneumatic section 400 includes an emergency application valve (EAV) 410, a service application valve (SAV) 420 and a release valve (RV) 430 along with an air spring transducer (AST) 450, a brake cylinder transducer (BCT) 460, a brake cylinder control transducer (BCCT) 470 and a trainline
25 pressure switch 480. The pneumatic section 600 features a variable load relay valve (VLRV) 601.

The universal unit 100 connects pneumatically within the 26-C Brake Control System 1 through several ports as shown in Figures 1 and 2. The input ports of universal unit 100 include a relay valve supply port 121, an air spring supply port 123, an emergency application supply port 124, a service application supply port 125 and a trainline pipe port 126. The output ports include a brake cylinder output port 122 and an exhaust port 127. The air spring supply port 123 receives air spring pressure from an air suspension system (not shown) on the rail vehicle in a manner well known in the brake control art. The service application supply port 125 receives a first source of pressure. The emergency application supply port 124 receives a second source of pressure and the relay valve supply port 121 receives a third source of pressure. In the 26-C System, the first and third sources of pressure are preferably the same, namely, the supply reservoir 2 of brake control system 1. The second source of pressure is a brake pipe 3 via the present invention, a BP sensing unit 800. Also, the trainline pipe port 126 receives pressure from the brake pipe in the 26-C System. The trainline or brake pipe pressure switch (BPPS) 480 then monitors the pressure in the brake pipe 3. In a manner well known in the brake control art, when pressure in brake pipe 3 drops to an emergency level, the BPPS 480 generates an electrical signal indicative of an emergency. The BPPS passes this emergency signal to the brake assurance means 300 where it is used in an emergency as described below.

The variable load relay valve 601 is an air piloted device whose construction and operation are well known in the brake control art. The VLRV 601 may optionally be replaced by a combination consisting of a variable load valve and a J-1 relay valve both of which are also well known. The VLRV 601 includes a supply port 651 connected to the third source of pressure via relay valve supply port 121, an output port 652 connected to the brake cylinders via brake cylinder output port 122, and a load weight port 653 connected to the air suspension system via air spring supply port 123 and a brake control port 654 connected both to the outputs of the EAV 410 and SAV 420 and to the input of the RV 430. The VLRV 601 also includes an exhaust port 655 through which pressure from the brake cylinders vents to atmosphere when the pressure at brake control port 654 falls below a minimal level in a manner well known in the art. The SAV 420 when open allows a low capacity service brake control pressure to pass from the first source of pressure to brake control port 654. The EAV 410 when open allows a low capacity emergency brake control pressure to pass from the second source of pressure to brake control port 654. The VLRV 601 responds to either low capacity brake control pressure by providing from output port 652 a high capacity pressure to the brake cylinders. Though the air supplied to the brake cylinders will obviously be provided in greater quantity than the air received at brake control port 654, the air delivered to the brake cylinders (i.e., brake cylinder pressure) and the air delivered to brake control port 654 (i.e., brake control pressures) will be

approximately equal in pressure as long as the brake cylinder pressure is below the air spring pressure provided to load weight port 653. Brake cylinder pressure is thus proportional to pressure that the VLRV 601 receives from the air suspension system. The
5 VLRV 601 uses air spring pressure to limit the maximum pressure at which air is directed to the brake cylinders. The VLRV 601 thus compensates for the weight of the load borne by the rail vehicle during both service and emergency brake applications.

Regarding the pressure transducers, the air spring
10 transducer (AST) 450 monitors the air spring pressure at the load weight port 653 of the VLRV 601. The brake cylinder control transducer (BCCT) 470 monitors the brake control pressure at the brake control port 654 of the VLRV. The brake cylinder transducer (BCT) 460 monitors the pressure supplied to the brake cylinders
15 from the output port 652 of the VLRV (i.e., the brake cylinder output port 22 of the universal unit).

The SAV, EAV and RV valves are each two-way magnet valves controlled by the electronic section 200 of universal unit 100 as indicated by Figures 2 and 3. The SAV 420 connects at its input
20 port to the first source of pressure via the service application supply port 125 and at its output port connects to the brake control port 654 of the VLRV 601. The EAV 410 connects at its input port to the second source of pressure via the emergency application supply port 124 and at its output port to the brake
25 control port 654 of the VLRV. The RV 430 connects at its input

port to brake control port 654 and vents to atmosphere through its exhaust port 127.

The universal unit 100 connects electrically within the 26-C Brake Control System 1 as best shown in Figure 3. Each rail vehicle usually includes a master electronic unit (MEU) (not shown) whose construction and operation are generally well known in the brake control art. The communication interface means 210 is the component through which universal unit 100 communicates with the MEU via a brake control bus 10 on the rail vehicle. The communication means 210 may take the form of any one of a variety of known devices used to communicate information between parts of an electronic system. Through brake control bus 10, the MEU communicates with and controls each of the universal units 100 on a given rail vehicle as explained below. Brake control bus 10 can be either an electronic or a fiber optic link and use any one of a variety of communication protocols known in the communication art.

The BP sensing unit 800 incorporates into the WABCO 26-C Brake Control System 1 along with the universal unit 100 as shown in Figure 1. BP sensing unit 800 is a novel device which serves at least in part as a pneumatic backup to the universal unit 100 on each of the trucks. BP sensing unit 800 pneumatically interconnects between the brake pipe 3 of brake control system 1 and emergency application supply port 124 of each universal unit 100 on the rail vehicle. As shown in Figure 1, BP sensing unit 800 connects to brake pipe 3 much like the prior art VX vent valve 4 and the prior art A-1 reducing relay valve 5 typically found on

many brake control systems. BP sensing unit 800 electrically connects within brake control system 1 through its BP transmission interface means 805. BP transmission means 805 may take the form of any one of a variety of known devices used to communicate information between parts of an electronic system. The transmission means connects BP sensing unit 800 to the MEU via brake control bus 10.

The present invention includes an auxiliary reservoir 802, a control reservoir 803, a brake pipe transducer (BPT) 804, the transmission interface means 805 and an MC-31 control valve portion 810. The MC-31 valve portion 810 includes a three state valve 811, a quick service valve (QSV) 812 and, optionally, a quick release valve (QRV) 813. The three state valve 811 is an air piloted valve whose construction and operation are generally well known in the brake control art. The three state valve includes two pilot ports and two exhaust ports E1 and E2. Its operation depends on the difference in pressure applied to its two pilot ports. As shown in Figure E, the first pilot port 821 receives pressure from brake pipe 3 while the second pilot port 822 receives pressure from control reservoir 803. The QSV 812 is an air piloted valve whose operation depends on the pressure applied to its service pilot port. The service pilot port pneumatically connects to the emergency application supply port 124 (i.e., the input of EAV 410) in each universal unit 100 on the rail vehicle. The QSV 812 remains in an open state as long as pressure at its pilot port remains below a preselected level as alluded to in Figure 1. The

QRV 813 is an optional feature of BP sensing unit 800. It is also an air piloted valve whose operation depends on the pressure applied to its release pilot port. The release pilot port pneumatically connects to port 840 of BP sensing unit 800. The QRV 5 813 remains in a closed state until the pressure at port 840 builds to the preselected or any other desired level. The auxiliary reservoir 802 also pneumatically connects to the emergency application supply ports 124. It essentially serves as a buffer against sharp increases and decreases in pressure.

10 The BP sensing unit 800 operates in response to the changes in pressure within the brake pipe 3 of the brake control system. Depending on the difference in pressure between the first and second pilot ports 821 and 822 of three state valve 811, the three state valve may assume any one of three positions: apply, lap
15 and release. When the 26-C System, for example, commands a decrease in brake pipe pressure to apply the brakes, the three state valve will soon move to the apply position. This occurs when the pressure in brake pipe 3 falls below that in the control reservoir 803 thereby dropping the pressure at first pilot port 821
20 relative to that at second pilot port 822. The three state valve then assumes the apply position in which pressurized air flows both to auxiliary reservoir 802 and to emergency application supply port 124 24 of each universal unit (i.e., to the input of the EAVs). This pressurized air flows to those two destinations through two
25 routes in BP sensing unit 800: (1) from control reservoir 803 through three state valve 811 and pipe 850 to the EAVs and (2) from

brake pipe 3 through three state valve 811 and the QSV 812 to the EAVs as long as the pressure at the pilot port of QSV 812 remains below the preselected level. The QSV advantageously diverts from
5 air that otherwise would be exhausted to atmosphere. When the pressure at service pilot port of QSV 812 exceeds the preselected level, the QSV closes thereby severing one connection between brake pipe 3 and emergency application supply ports 124. Once the QSV closes, the QSV prevents the pressurized air supplied to the EAVs
10 from flowing back into brake pipe 3. During brake applications, the QSV thus assists not only in quickly reducing pressure in brake pipe 3 but also in quickly increasing the pressure supplied to the emergency application supply port 124 of each universal unit.

When the 26-C System, for example, commands an increase
15 in brake pipe pressure to release the brakes, three state valve 811 will soon move to the release position. This occurs when the pressure in brake pipe 3 exceeds the pressure in control reservoir 803 thereby increasing the pressure at first pilot port 821 relative to that at second pilot port 822. The three state valve
20 then assumes the release position in which pressurized air from brake pipe 3 flows through three state valve 811 to pressurize control reservoir 803. Simultaneously, BP sensing unit 800 vents to atmosphere pressurized air from both auxiliary reservoir 802 and the emergency application supply port 124 of each universal unit.
25 It vents such pressurized air from those two sources to atmosphere through two routes in BP sensing unit 800: (1) from the EAVs

through pipe 850 and three state valve 811 to atmosphere via exhaust port E1 and (2) from the EAVs through QSV 812 and three state valve 811 to atmosphere via exhaust port E2. The latter route is available only when the pressure at the pilot port of QSV 812 again drops below the preselected level.

The optional QRV 813 may be used to pressurize the brake pipe even more quickly when the 26-C System 1 commands an increase in brake pipe pressure to release the brakes. Specifically, as brake pipe 3 begins to charge, the QRV 813 may be used to provide a route through which pressurized air remaining in control reservoir 803 can flow back into brake pipe 3. During brake pipe charging, the release pilot port of QRV 813 need only receive a pressure sufficient to open the QRV. Once open, the QRV would allow the pressurized air to flow from control reservoir 803 through pipe 850 back into brake pipe 3 as shown in Figure 1. Functioning in a manner similar to that of QSV 812, the QRV 813 may be used to assist in quickly recharging the brake pipe and releasing the brakes.

When the control reservoir eventually reaches an equilibrium with the brake pipe, the pressures respectively applied to the first and second ports 821 and 822 of three state valve 811 equalize. The three state valve then assumes the lap position in which the pressurized air from brake pipe 3 flows through a choke in three state valve 811 to keep control reservoir 803 pressurized. The three state valve thus maintains within control reservoir 803 the pressure that will be needed to pneumatically instigate the

next brake application via the universal units. Simultaneously, BP sensing unit 800 still vents to atmosphere pressurized air from both auxiliary reservoir 802 and the emergency application supply port 124 of each universal unit. As shown in Figure 1, however, it vents such pressurized air from those two sources to atmosphere only through one route in the BP sensing unit: from the EAVs through QSV 812 and three state valve 811 to atmosphere via exhaust port E2. As long as the pressure at its pilot port remains below the preselected level, the QSV remains open and allows such pressurized air to vent to atmosphere.

The BP sensing unit may also feature a CS magnet valve 830. The CS magnet valve 830 is a valve whose construction and operation are generally well known in the brake control art. It may or may not be incorporated into the BP sensing unit depending into which particular brake control system the invention will be incorporated. In the WABCO 26-C CS-2 Style Brake Control System, for example, BP sensing unit 800 may feature CS magnet valve 830 as shown in Figure 1. The CS magnet valve could be used to vent brake pipe 3 to atmosphere when energized via an electrical trainline. When the brake control system 1 commands a decrease in brake pipe pressure to apply the brakes, for example, it could also energize CS magnet valve 830 so that brake pipe pressure would drop even more quickly. CS magnet valve 830 can therefore be used to assist in quickly decreasing pressure in brake pipe 3 and applying the brakes.

It should be apparent to persons skilled in the brake control art that the BP sensing unit may conceivably be employed on brake control systems other than the 26-C discussed above. Obvious modifications may be necessary, though, such as changing the pneumatic trainline to which the sensing unit would connect. This would, of course, depend upon the specific application in which the sensing unit is employed.

Regarding the operation of the present invention, the WABCO 26-C Brake Control System through its central controller unit (not shown) sends the brake command signal pneumatically along brake pipe 3 to each of the rail vehicles. It is through this brake command signal that the train operator ultimately controls the brakes of the rail vehicles in the train. The brake pipe transducer (BPT) 804 of BP sensing unit 800 on each rail vehicle converts the pneumatic brake command signal into an electrical signal which it passes to the BP interface means 805. Through BP transmission means 805, BP sensing unit 800 makes this electrical brake command signal available to the MEU or any other devices on brake control bus 10. The MEU receives not only the brake command signal but also other signals through which it controls each of the trucks on the rail vehicle.

The MEU in each rail vehicle performs the Brake Control Process depicted in blocks 1-15 of Figure 4. This process is somewhat different than the prior art brake control process described above especially as it pertains to the functions illustrated in blocks 15A and 15B. As shown in Figure 4, the MEU

receives various signals and determines a brake force command signal according to principles well known in the brake control art. The brake force command signal is an electrical signal ultimately indicative of the pressure that ought to be supplied to the brake cylinders of each truck. It is communicated to each universal unit as indicated in blocks 15A and 15B.

The MEU, for example, reads the rate request signal, the power/brake signal and the emergency brake signal received from the brake control system as shown in block 1. In certain brake control systems such as the 26-C System, the rate request signal and the emergency brake signal are both conveyed to the MEU through the same route, i.e., along brake pipe 3 to BPT 804 to the MEU, and are essentially one and the same. In other brake control systems, they may be conveyed to the MEU through different routes such as separate electrical and pneumatic trainlines. Depending on the type of brake control system in which the present invention is employed, the rate request signal may be representative not only of the brake command signal but also the rate at which to propel the rail vehicle during propulsion. In the 26-C System, however, the rate request signal represents the brake command signal. The power/brake signal is used in conjunction with the rate request signal to distinguish whether the train is in the braking mode or the propulsion mode.

Block 1 of the Brake Control Process illustrates that the MEU uses these signals to determine the rate at which the entire train is to be braked, i.e., train braking rate request (in mphps).

Block 2 illustrates that the MEU converts the train braking rate request into the rates at which the trucks on the rail vehicle are to be braked, i.e., the truck braking rate request (in mphps). The truck braking rate request represents the amount of braking force that should be applied by the brakes of each truck.

The MEU may also process other signals such as a speed signal, an air spring pressure signal, a dynamic brake feedback signal, a snow brake request signal and a wheel slip control signal. Block 3 of the Brake Control Process illustrates that the MEU may use the speed signal to perform speed tapering, an optional feature of the Brake Control Process. The output of block 3 is a modified truck braking request. Block 4 represents jerk limiting through which to reduce the jerking that would otherwise be encountered with quick acceleration or braking. The rate at which the truck brakes are applied or the rate at which the rail vehicle is accelerated may be limited to the preset jerk limit (mphpsps). The jerk limited truck braking request passes to block 6.

Blocks 5 and 6 of the Brake Control Process show that the air spring pressure signal may be used to modify the jerk limited truck braking request so as to compensate for the weight borne by the rail vehicle during brake applications. Based on rail vehicle weight and the jerk limited truck braking request, the MEU calculates the required friction tractive effort to be applied by the brakes of each truck as indicated by block 6. Block 7 and 8 show that the wheel slip control can be used to reduce the friction tractive effort in response to wheel slippage. Blocks 9-12 show

that the friction tractive effort may be further reduced by the amount of dynamic braking effort provided by the propulsion units.

Block 13 illustrates that MEU may also take into account the snow brake request. The friction tractive effort may be further reduced to account for the snow braking effort already applied to the wheels. The output of block 13 represents the final braking effort to be applied to the wheels of each truck. Block 14 illustrates that the MEU converts the final friction braking effort into an electrical signal indicative of brake cylinder pressure. Equivalent to the final braking effort, the brake cylinder pressure signal represents the amount of pressure that ought to be supplied to the brake cylinders of each truck of the rail vehicle. Blocks 15A and 15B each convert the brake cylinder pressure signal into the type of voltage or current signal that can be processed by each of the universal units 100 on the rail vehicle. This signal, called the brake force command signal, is sent to each universal unit 100 as shown in blocks 15A and 15B.

It should be apparent to persons skilled in the brake control art that each universal unit 100 could be designed to perform part or all of the Brake Control Process in addition to those tasks described below. The various signals needed to perform the process would be conveyed directly or indirectly to universal unit 100 via the trainlines or other means as shown in Figure 3. The MEU could then serve as a backup or perform more of a communication function or other desired tasks or both. Obvious

modifications may be necessary, though, depending upon the specific application in which the universal unit is employed.

Referring now to Figures 2 and 3, the MEU sends the brake force command signal to the communication interface means 210 of each universal unit on the rail vehicle. Communication means 210 conveys the brake force command signal to local control means 230. Local control means 230 is basically a computing device through which universal unit 100 controls its magnet valves (i.e., the RV, SAV and EAV) during most conditions as explained below. Specifically, in response to the brake force command signal, local control means 230 generates valve command signal(s) to direct solenoid driver means 250 to energize the appropriate magnet valve(s). For reasons that will soon be apparent the valve commands sent from local control means 230 to solenoid driver means 250 are routed through brake assurance means 300.

The brake force command signal can also be referred to as a command for brake control pressure because it represents the pressure that should be supplied to the brake control port 654 of VLRV 601. The brake force command signal, of course, is also ultimately indicative of the pressure that should be supplied to the brake cylinders of each truck. Local control means 230 receives not only the command for brake control pressure but also feedback in the form of a brake control pressure feedback signal. The brake control pressure feedback signal is indicative of the pressure currently at brake control port 654 of VLRV 601. It is

sent from the BCCT 470 via transducer interface means 260 and communication interface means 210.

The local control means essentially controls the pressure that is applied to brake control port 654 of the VLRV. Local control means 230 executes a summing point function through which it compares the command for brake control pressure received ultimately from the MEU with the brake control pressure feedback signal received ultimately from BCCT 470. If the brake control pressure feedback signal represents a pressure greater than that represented by the command for brake control pressure, local control means 230 will command solenoid driver means 250 to energize (open) RV 430 either continuously or pulsingly to reduce the pressure at brake control port 654 of the VLRV to the desired level. Local control means 230 will also simultaneously command solenoid driver means 250 to direct SAV 420 to its normally closed (deenergized) state and EAV 410 remains in its closed (energized) state. This prevents service brake control pressure or emergency brake control pressure from affecting the pressure at brake control port 654 of the VLRV. If the brake control pressure feedback signal represents a pressure whose value lies within a preset band around that represented by the command for brake control pressure, local control means 230 will command solenoid driver means 250 to maintain the pressure currently prevailing at brake control port 654 of the VLRV. Solenoid driver means 250 will thus direct the SAV and RV to their closed (deenergized) states and the EAV remains in its closed (energized) state. The pressure at brake control

port 654 thus remains constant. If the brake control pressure feedback signal represents a pressure less than that represented by the command for brake control pressure, the local control means will command the solenoid driver means to energize (open) the SAV either continuously or pulsingly to increase the pressure at brake control port 654. Local control means 230 will also simultaneously command the solenoid driver means to direct the RV to its normally closed (deenergized) state and the EAV remains in its closed (energized) state. This allows service brake control pressure to build at control port 654 of the variable load relay valve 601.

The VLRV responds to either brake control pressure by providing a corresponding pressure to the brake cylinders that is proportional to pressure that VLRV 601 receives from the air suspension system. This yields a final friction braking effort to the wheels of the truck that compensates for the weight of the load borne by the rail vehicle. Local control means 230 responds in this manner continuously to both the command for brake control pressure and the brake control pressure feedback signal as these commands are obviously subject to continuous change.

It should be apparent that universal unit 100 may optionally use a J-1 relay valve instead of variable load relay valve 601 depending on the particular train at issue. Certain 26-C Systems, for example, do not require the VLRV even though each of the rail vehicles of the train has an air suspension system. This is because the difference between the weight of the railcar and that of the load may be insufficient to warrant the addition of the

load compensating function of the VLRV. The J-1 relay valve would be incorporated into the universal unit in a manner identical to the VLRV with the exception of load weight port 653.

5 The local control means may also perform certain other tasks which can be performed according to principles well known in the brake control art. For example, local control means 230 can be used to determine whether the speed sensors are operating properly. This includes determining whether any discontinuities have developed in the associated wiring. It does this through a speed
10 sensor test signal which it sends via the speed sensor interface means 240 to each speed sensor located on the axles of the truck. The result signal generated by each speed sensor in response to the test signal is compared to the test signal. The speed sensors operate properly if each result signal comports with the test
15 signal. Whatever the result of the comparison, local control means 230 sends a diagnostic message indicative of the state of each speed sensor to communication means 210. It is through communication means 210 that universal unit 100 can make the diagnostic message available to any other devices on brake control
20 bus 10. The system may be configured so that either the MEU commands local control means 230 to check the operation of the speed sensors or that local control means 230 does so unilaterally.

Each truck axle is equipped with a speed sensor as is well known in the brake control art. The speed sensor interface
25 means 240 receives input from these speed sensors and converts them into signals indicative of speed and the rate at which the rail

vehicle is accelerating or decelerating. Speed sensor means 240 sends these speed and accel/decel signals to communication means 210 through which universal unit 100 can make them available to other devices on brake control bus 10 including the MEU and the
5 brake assurance means 300 as discussed in greater detail below. The MEU may use the speed signal (e.g., in speed tapering) and the accel/decel signal (e.g., in jerk limiting) to calculate the final friction braking effort according to the Brake Control Process outlined in Figure 4.

10 The transducer interface means performs tasks which can be performed according to principles well known in the brake control art. Specifically, transducer interface means 260 supplies power to each of the transducers in the universal unit such as the brake cylinder transducer (BCT) 460, the brake cylinder control
15 transducer (BCCT) 470, and the air spring pressure transducer (AST) 450. The BCT supplies an electrical brake cylinder pressure feedback signal indicative of the pressure supplied to the brake cylinders of the truck. The BCCT supplies the electrical brake control pressure feedback signal indicative of the pressure applied
20 to brake control port 654 of the VLRV. The ACT supplies the electrical air spring pressure signal indicative of the air spring pressure that it receives from the air suspension system. Transducer interface means 260 conditions these signals and sends one or more of them to brake assurance means 300 and to
25 communication means 210. Through communication means 210, universal unit 100 can make these signals available to any of its

components or to any other devices on brake control bus 10 including the MEU.

The solenoid driver means contains the solenoid driver circuits responsible for driving the emergency application valve (EAV), the service application valve (SAV) and the release valve (RV). These driver circuits receive the valve command signals from local control means 230 through which it drives these magnet valves. The valve command signals from local control means 230 are allowed to pass through brake assurance means 300 to solenoid driver means 250 during normal operating conditions. If the brakes do not apply during normal service or emergency braking, brake assurance means 300 will interrupt all communication between local control means 230 and solenoid driver means 250 so that only brake assurance means 300 can control the magnet valves as explained below.

During normal service or emergency braking, when solenoid driver means 250 receives a release valve command signal from local control means 230, it energizes (opens) the RV either continuously or pulsingly to reduce the pressure at brake control port 654 of VLRV 601. It also simultaneously directs the SAV to its normally closed (deenergized) state and the EAV remains commanded to its closed (energized) state. This prevents service brake control pressure or emergency brake control pressure from reaching brake control port 654. When solenoid driver means 250 receives a service valve command signal from local control means 230, it energizes (opens) the SAV either continuously or pulsingly to

increase the pressure at brake control port 654. It also simultaneously directs the RV to its normally closed (deenergized) state and the EAV remains commanded to its closed (energized) state. This allows service brake control pressure to build at
5 brake control port 654. When solenoid driver means 250 does not receive an emergency valve command signal from local control means 230, it deenergizes (opens) the EAV to increase the pressure at brake control port 654. It also simultaneously directs the RV to its normally closed (deenergized) state. This allows emergency
10 brake control pressure to build at brake control port 654. Variable load relay valve 601 responds to each brake control pressure in the previously described manner.

The solenoid driver means also performs certain other tasks which can be performed according to principles well known in
15 the brake control art. For example, feedback from the coils of each of the two-way valves may be supplied to solenoid driver means 250 and passed to communication means 210 of universal unit 100 as shown in Figure 3. Universal unit 100 can make these valve feedback signals available to any of its components or to any other
20 devices on brake control bus 10.

The valve feedback signals may be used by the overall system in any number of ways. They may, for example, be used for diagnostic purposes to determine whether any one or more of the magnet valves actually responded to the valve command signals
25 issued by local control means 230. Specifically, from the valve feedback signals the MEU could determine whether the magnet valves

correctly responded to the command for brake control pressure (i.e., brake force command signal) it sent to local control means 230. If the valve feedback signals do not comport with the brake force command signal issued by the MEU, the MEU can pass a diagnostic or status message indicative of valve malfunction or other failure to the central controller unit of the brake control system into which the present invention is incorporated. The central controller unit can, of course, be programmed to react in whatever manner the train authority deems appropriate. Through a visual or audio indication, for example, the brake control system can at a minimum inform the train operator of the exact nature of the problem.

Referring now to Figure 5, brake assurance means 300 includes logic OR and AND gates 301 and 302, an emergency relay (EMRL) driver 310, emergency (EMRL) contacts, a summing circuit 320, a level detector circuit 330, a delay circuit 340, a timer circuit 350, a valve relay driver 360 and normally open solenoid relay contacts SR1 and SR2. The primary inputs to brake assurance means 300 are the full service demand signal and the emergency signal. These signal are supplied by the brake control system into which the brake assurance means 300 is incorporated.

The full service demand signal merely represents a particular brake command signal (i.e., a particular rate request signal) through which the brake control system requests full application of the brakes. The full service demand signal may be routed from the brake control system via an electrical trainline

either (1) to and through the MEU, brake control bus 10 and communication interface 210 into brake assurance means 300 (as in RT-5 T-1 Style and RT-5 MARTA Style) or (2) to and through communication interface 210 into brake assurance means 300 or (3) directly into brake assurance means 300. Similarly, the full service demand signal may be routed from the brake control system via a pneumatic trainline to a pressure transducer ultimately through the MEU, brake control bus 10, communication interface 210 and into brake assurance means 300. In the 26-C Brake Control System, for example, the central controller unit would send the full service demand signal pneumatically along brake pipe 3. The BPT 804 of BP sensing unit 800 would convert the pneumatic full service demand signal into an electrical signal which it passes to BP interface means 805. BP sensing unit 800 makes this electrical signal available to the MEU which passes it along brake control bus 10 to communication means 210 and ultimately into brake assurance means 300.

The EMRL driver 310 of brake assurance means 300 receives the emergency signal from the brake control system. Typically, the emergency signal is routed via (1) an electrical trainline 7 directly into brake assurance means 300 or (2) a pneumatic trainline into a pressure switch and then into the brake assurance means. In the 26-C Brake Control System, for example, the central controller unit would send the emergency signal pneumatically along brake pipe 3. When pressure in brake pipe 3 drops to an emergency level, the BPPS 480 of each universal unit 100 generates an

electrical signal indicative of the emergency. The BPPS 480 then passes this emergency signal to brake assurance means 300. The EMRL driver 310 reduces the voltage level of the emergency signal so that it is made compatible with the logic circuitry of brake assurance means 300. EMRL driver 310 also feeds back this logic emergency detect signal to communication means 210 as is shown in Figures 3 and 5. Both the full service demand signal and the emergency signal are applied to the input of OR gate 301 whose output is fed into the trigger/enable input of timer circuit 350. The full service demand signal, of course, may or may not be used in this arrangement.

The summing circuit 320 receives the brake cylinder pressure signal via a first feed line and the dynamic brake feedback signal via a second feed line and sums these signals into a total braking effort signal. The former signal comes from the BCT 460 via transducer interface 260 and represents the actual friction braking effort of the brakes of the truck. The latter signal ultimately comes from the propulsion unit(s) and represents the dynamic braking effort applied through the propulsion motors.

The level detector circuit 330 generates a threshold signal when the total braking effort signal received from summing circuit 320 exceeds a predetermined threshold. The delay circuit 340 generates a level signal after having received the threshold signal throughout a preset time period. The emergency relay driver 310 controls a first emergency contact in series within the second feed line. When emergency relay driver 310 receives an emergency

signal from the brake control system, it conveys an emergency detect signal and opens a first emergency contact thereby preventing the dynamic braking effort signal from contributing to the total braking effort signal. It opens simultaneously a second emergency contact thereby preventing feedback concerning the state and condition of the EMRL contacts from reaching the communication means 210.

The valve relay driver 360 controls the opening and closing of the SR1 and SR2 contacts. Two of the SR1 contacts are in series with the power and return control lines that feed the coil of the emergency application valve 410. Another two of the SR1 contacts are in series with a status line 411 feeding into communication means 210. Likewise, two of the SR2 contacts are in series with the power and return control lines that feed the coil of the release valve 430. Another two of the SR2 contacts are in series with a status line 431 feeding into communication means 210. Through these two solenoid status lines, brake assurance means 300 passes feedback concerning the state of the valves as well as the state and condition of the SR1 and SR2 contacts to communication means 210. Valve relay driver 360 when activated closes the solenoid contacts thereby permitting local control means 230 to control the emergency application and release magnet valves 410 and 430. When deactivated, valve relay driver 360 opens the solenoid contacts thereby preventing the local control means 230 from controlling the EAV and RV valves.

The timer circuit 350 has a set input, a trigger input and a reset input. When the set input receives an indication that both the level signal and a zero speed signal occur simultaneously, timer circuit 350 activates valve relay driver 360 thereby placing
5 brake assurance means 300 in a standby mode of operation. During the standby mode, timer circuit 350 monitors the trigger input for the emergency detect signal and a full service demand signal in response to either or both of which timer circuit 350 becomes enabled. If the reset input receives the level signal within a
10 predetermined time after timer circuit 350 has become enabled then timer circuit 350 resets thereby keeping brake assurance means 300 in the standby mode. Otherwise, timer circuit 350 deactivates valve relay driver 360 thereby placing brake assurance means 300 in an intervention mode of operation.

15 The brake assurance means 300 operates in the intervention mode in response to the predetermined set of conditions set forth above or in response to a loss of power. Once brake assurance means 300 is placed into the intervention mode, it will not return to the standby mode and reactivate valve relay
20 driver 360 until timer circuit 350 receives the set signal at its set input.

Certain authorities in the train industry require that wheel slip control be inhibited during emergency brake applications. Brake assurance means 300 may therefore include two
25 wheel slip inhibit (WSI) contacts as shown in Figure 5. One WSI contact is in series with the power line that feeds the coil of

emergency application valve 410. The other WSI contact is in series with the power line that feeds the coil of release valve 430. Either the MEU or universal unit 100 may control the opening and closing of the WSI contacts as noted previously.

5 Should wheel slip control be provided, the MEU via local control means 230 would at the very least briefly open (energize) release valve 430 during wheel slippage to reduce the pressure in the brake cylinders in the previously described manner. This would reduce the braking force being applied to the wheels of the truck
10 to the point where the wheels are no longer sliding along the rails. When the wheels stop sliding, the MEU via the universal units would again bring the braking force up to the level requested by the brake control system via the brake command signal.

15 The wheel slip control function, however, may be inhibited during emergency braking by opening the WSI contacts in the brake assurance means. With the WSI contacts open, the EAV and RV will each assuredly be deenergized (open and closed, respectively). This, of course, allows emergency brake control pressure to continue to build at brake control port 654 of variable
20 load relay valve 601. VLRV 601 responds accordingly by providing to the brake cylinders the corresponding load compensated brake cylinder pressure. The brakes respond by applying the emergency braking force to the wheels of the truck.

25 The WSI contacts come into play while brake assurance means 300 operates in the standby mode and the MEU via local control means 230 commands an emergency brake application. By

opening the WSI contacts during these conditions, wheel slip control can be inhibited during emergency braking. It should be apparent that the wheel slip control may optionally be inhibited during service braking.

5

While the presently preferred embodiment and various related aspects of the instant invention have been set forth in detail according to the Patent Act, those persons of ordinary skill in the technical art to which this invention pertains will recognize various alternative ways of practicing the invention without departing from the spirit and scope of the appended claims. Those of ordinary skill will also recognize that the foregoing description is merely illustrative and is not intended to limit any of the following claims to any particular narrow interpretation.

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Accordingly, to promote the progress of science and useful arts, we secure for ourselves by Letters Patent exclusive rights to all subject matter embraced by the following claims for a time prescribed by the Patent Act.



THE CLAIMS DEFINING THE INVENTION ARE AS FOLLOWS:-

1. A brake pipe sensing unit for use with a universal pneumatic brake control unit of a rail vehicle, said brake pipe sensing unit comprising:

(a) a control reservoir within which gas under pressure may be stored and from which gas under pressure may be supplied;

(b) an auxiliary reservoir in communication with a supply port of said universal unit and with a first and a second route of said brake pipe sensing unit;

(c) a three state valve having a first pilot port in communication with a brake pipe of said rail vehicle, a second pilot port in communication with said control reservoir, and first and second exhaust ports both of which connect to atmosphere, said three state valve for operating in:

(i) an apply position when pressure at said first pilot port drops relative to pressure at said second pilot port thereby allowing flow from said brake pipe through said three state valve and said first route both to said auxiliary reservoir and to said supply port and from said control reservoir through said three state valve and said second route both to said auxiliary reservoir and to said supply port;

(ii) a release position when pressure at said first pilot port increases relative to pressure at said second pilot port thereby allowing flow from said brake pipe through said three state valve to said control reservoir and from both said

auxiliary reservoir and said supply port through said first and said second routes through said three state valve to atmosphere via said first and said second exhaust ports; and

(iii) a lap position when pressure at said first pilot port equalizes relative to pressure at said second pilot port thereby allowing flow from said brake pipe to said control reservoir and from both said auxiliary reservoir and said supply port through said first route and said three state valve to atmosphere via said second exhaust port;

(d) a quick service valve connected within said first route and having a service pilot port in communication with both said auxiliary reservoir and said supply port such that said quick service valve closes said first route when pressure at said service pilot port exceeds a preselected level;

(e) a brake pipe transducer for converting pressure within said brake pipe to an electrical brake command signal; and

(f) a transmission means for communicating said brake command signal received from said brake pipe transducer to a brake control bus on said rail vehicle.

2. The brake pipe sensing unit recited in claim 1 wherein said transmission means communicates said brake command signal via said brake control bus to a master electronic unit on said rail vehicle for use in a brake control process.

3. The brake pipe sensing unit recited in claim 1 wherein said transmission means communicates said brake command signal via said brake control bus to said universal unit for use in a brake control process.

5

4. The brake pipe sensing unit recited in claim 1 further including a quick release valve having a release pilot port such that said quick release valve allows communication between said brake pipe and said control reservoir when pressure at said release pilot port exceeds a desired level.

10

5. The brake pipe sensing unit recited in claim 1 further including a CS magnet valve for assisting when energized in venting pressure within said brake pipe to atmosphere.

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6. The brake pipe sensing unit recited in claim 1 wherein said supply port is an emergency application supply port of said universal unit.

7. The brake pipe sensing unit recited in claim 1 wherein said second route includes a pipe interconnecting said three state valve with both said auxiliary reservoir and said universal unit.

20

8. A pneumatic trainline sensing unit for use on a rail vehicle within a brake control system, said pneumatic trainline sensing unit comprising:

5 (a) a control reservoir within which gas under pressure may be stored and from which gas under pressure may be supplied;

(b) an auxiliary reservoir in communication with a supply port of said brake control system and with a first and a second route of said sensing unit;

10 (c) a three state valve having a first pilot port in communication with a pneumatic trainline of said rail vehicle, a second pilot port in communication with said control reservoir, and first and second exhaust ports both of which connect to atmosphere, said three state valve for operating in:

15 (i) an apply position when pressure at said first pilot port drops relative to pressure at said second pilot port thereby allowing flow from said pneumatic trainline through said three state valve and said first route both to said auxiliary reservoir and to said supply port and from said control reservoir through said three state valve and said second route both to said auxiliary reservoir and to said supply port;

20 (ii) a release position when pressure at said first pilot port increases relative to pressure at said second pilot port thereby allowing flow from said pneumatic trainline through said three state valve to said control reservoir and from both said auxiliary reservoir and said supply port through said

25

first and said second routes through said three state valve to atmosphere via said first and said second exhaust ports; and

(iii) a lap position when pressure at said first pilot port equalizes relative to pressure at said second pilot port thereby allowing flow from said pneumatic trainline to said control reservoir and from both said auxiliary reservoir and said supply port through said first route and said three state valve to atmosphere via said second exhaust port;

(d) a quick service valve connected within said first route and having a service pilot port in communication with both said auxiliary reservoir and said supply port such that said quick service valve closes said first route when pressure at said service pilot port exceeds a preselected level;

(e) a pneumatic trainline transducer for converting pressure within said pneumatic trainline to an electrical brake command signal; and

(f) a transmission means for communicating said brake command signal received from said pneumatic trainline transducer to a brake control bus on said rail vehicle.

9. The pneumatic trainline sensing unit recited in claim 8 wherein said transmission means communicates said brake command signal via said brake control bus to a master electronic unit for use in a brake control process.

10. The pneumatic trainline sensing unit recited in claim 8 wherein said transmission means communicates said brake command signal via said brake control bus to a universal pneumatic brake control unit for use in a brake control process.

5 11. The pneumatic trainline sensing unit recited in claim 8 further including a quick release valve having a release pilot port such that said quick release valve allows communication between said pneumatic trainline and said control reservoir when pressure at said release pilot port exceeds a desired level.

10 12. The pneumatic trainline sensing unit recited in claim 8 further including a CS magnet valve for assisting when energized in venting pressure within said pneumatic trainline to atmosphere.

15 13. The pneumatic trainline sensing unit recited in claim 8 wherein said supply port is an emergency application supply port of a universal pneumatic brake control unit.

20 14. The pneumatic trainline sensing unit recited in claim 8 wherein said second route includes a pipe interconnecting said three state valve with both said auxiliary reservoir and a universal pneumatic brake control unit.

15. A brake pipe sensing unit for use with a universal pneumatic brake control unit of a rail vehicle, said brake pipe sensing unit comprising:

5 (a) a control reservoir within which gas under pressure may be stored and from which gas under pressure may be supplied;

(b) an auxiliary reservoir in communication with a supply port of said universal unit and with a first and a second route of said brake pipe sensing unit;

10 (c) a three state valve having a first pilot port in communication with a brake pipe of said rail vehicle, a second pilot port in communication with said control reservoir, and first and second exhaust ports both of which connect to atmosphere, said three state valve for operating in an apply position, a release position and a lap position depending on the pressure bearing on
15 said first and second pilot ports;

(d) a quick service valve connected within said first route and having a service pilot port in communication with both said auxiliary reservoir and said supply port such that said quick service valve closes said first route when pressure at said service
20 pilot port exceeds a preselected level;

(e) a brake pipe transducer for converting pressure within said brake pipe to an electrical brake command signal; and

(f) a transmission means for communicating said brake command signal received from said brake pipe transducer to a brake control
25 bus on said rail vehicle.

16. The brake pipe sensing unit recited in claim 15 wherein said three state valve operates in:

5 (a) said apply position when pressure at said first pilot port drops relative to pressure at said second pilot port thereby allowing flow from said brake pipe through said three state valve and said first route both to said auxiliary reservoir and to said supply port and from said control reservoir through said three state valve and said second route both to said auxiliary reservoir and to said supply port;

10 (b) said release position when pressure at said first pilot port increases relative to pressure at said second pilot port thereby allowing flow from said brake pipe through said three state valve to said control reservoir and from both said auxiliary reservoir and said supply port through said first and said second routes through said three state valve to atmosphere via said first and said second exhaust ports; and

15 (c) said lap position when pressure at said first pilot port equalizes relative to pressure at said second pilot port thereby allowing flow from said brake pipe to said control reservoir and from both said auxiliary reservoir and said supply port through said first route and said three state valve to atmosphere via said second exhaust port.

20 25 17. The brake pipe sensing unit recited in claim 15 wherein said transmission means communicates said brake command

signal via said brake control bus to a master electronic unit on said rail vehicle for use in a brake control process.

18. The brake pipe sensing unit recited in claim 15
5 wherein said transmission means communicates said brake command signal via said brake control bus to said universal unit for use in a brake control process.

19. The brake pipe sensing unit recited in claim 15
10 further including a quick release valve having a release pilot port such that said quick release valve allows communication between said brake pipe and said control reservoir when pressure at said release pilot port exceeds a desired level.

20. The brake pipe sensing unit recited in claim 15
15 further including a CS magnet valve for assisting when energized in venting pressure within said brake pipe to atmosphere.

21. The brake pipe sensing unit recited in claim 15
20 wherein said supply port is an emergency application supply port of said universal unit.

22. The brake pipe sensing unit recited in claim 15
25 wherein said second route includes a pipe interconnecting said three state valve with both said auxiliary reservoir and said universal unit.

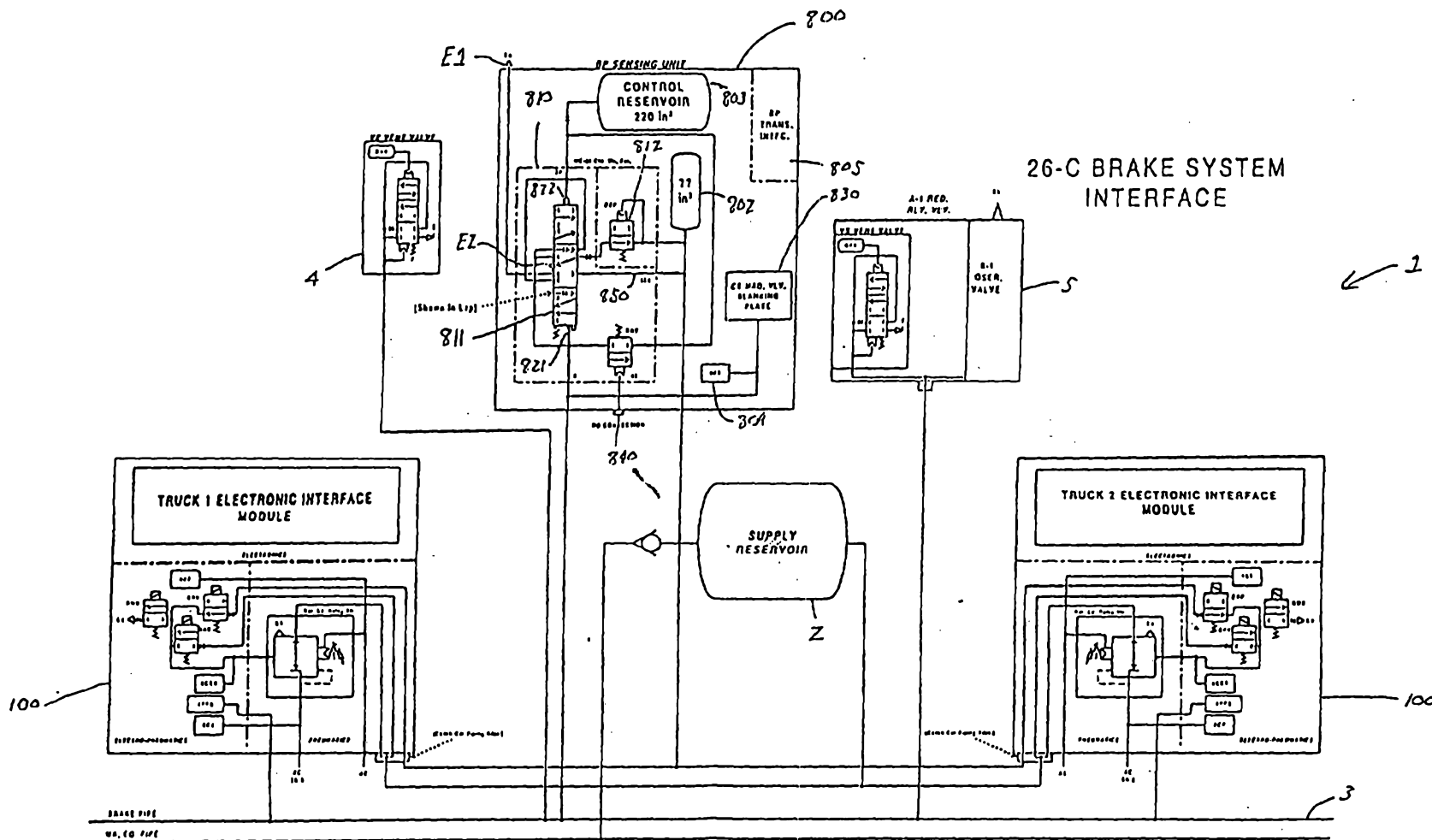
23. A brake pipe sensing unit for use with a universal pneumatic brake control unit of a rail vehicle as hereinbefore particularly described with reference to what is shown in Fig. 1 or Figs 2 to 5.

DATED: 23 January 1998

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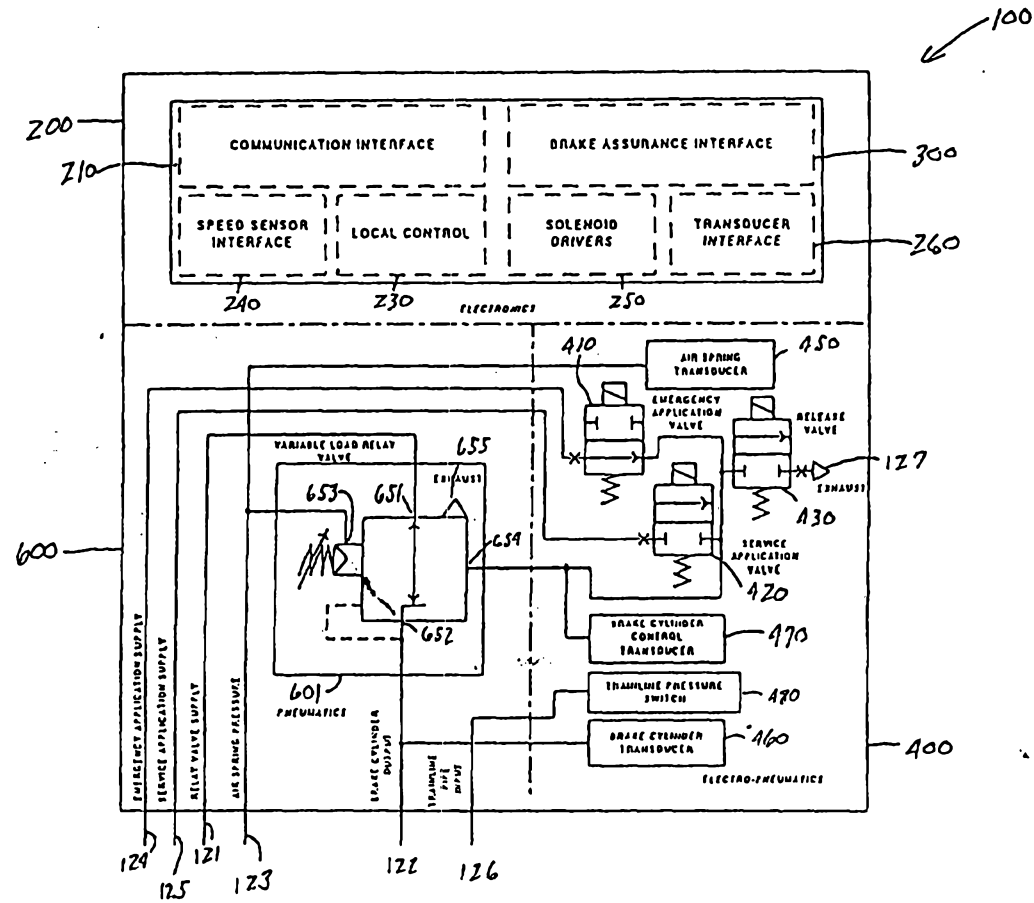
FIGURE 1



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20 01 00 5075

FIGURE 2



20 01 00 0725

FIGURE 3

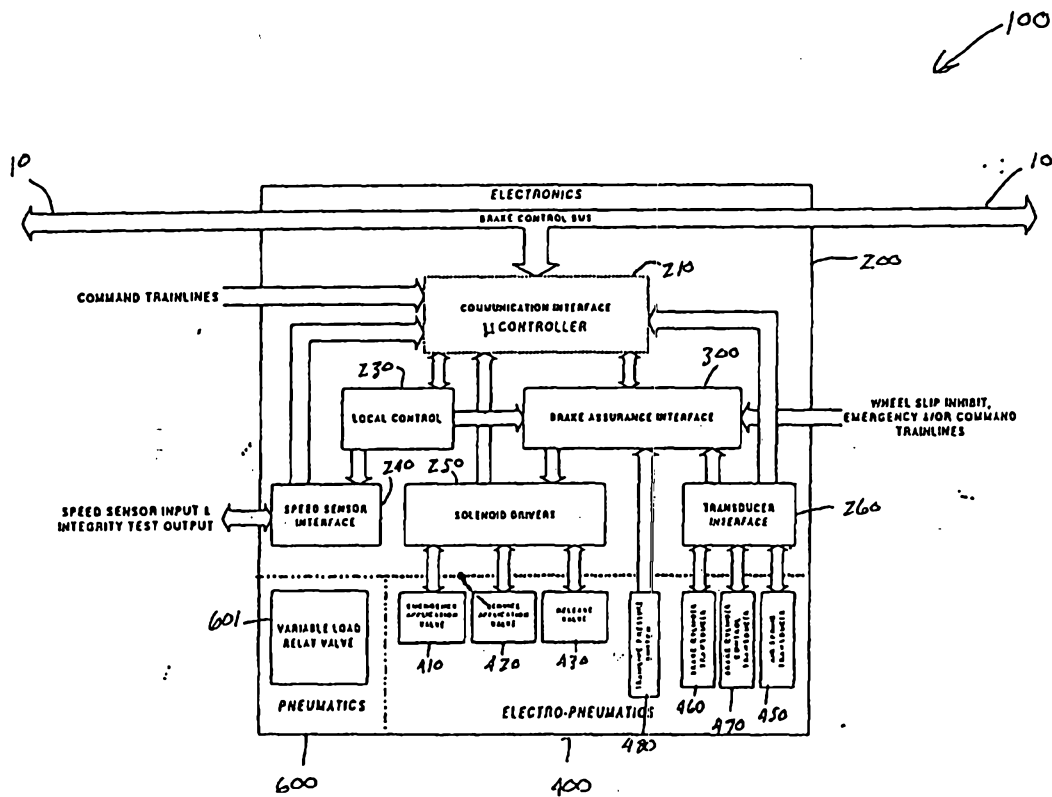


FIGURE 4

