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**EP-A1- 0 141 570**  
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**US-A- 3 791 628**  
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**US-A- 4 886 397**



# DESCRIPTION

**[0001]** The present invention relates to a device for use on an offshore vessel, more precisely the invention relates to a device for use in connection with an active wave- or heave compensator. Compensator systems like that are often arranged together with fixed passive compensators at the top of a derrick or at the bottom of the derrick near the hoisting dead end on a floating drilling installation.

## Background of the invention

**[0002]** An active heave compensator usually comprises a linear actuator, typically a hydraulic cylinder construction with three chambers. This construction has typically axial movement of about 8 meters. The actuator could also be of another form of linear actuator, such as electric powered. This active compensator actuator is generally related to a passive heave compensation machine, called drill string compensator or passive compensator. The passive compensator exploits the gas compressibility of air or nitrogen in large pressurized containers (APV's) which are hydraulically connected to one or more hydraulic cylinders via piston accumulator(s). The piston accumulator(s) is acting as a phase separator between the pressurized gas volume and the oil-filled hydraulic cylinders. The hydraulically cylinders balance thereof the load in this passive compensator and offsets most of the vessel heave motions using spring force from the pressurized gas volume. A typical drill string compensator manipulates the load in that the hydraulic cylinders in the passive compensator are mechanically connected to a movable crown block in the draw work system of the derrick. The actuator of the active compensator is also connected to and manipulates actively the movable crown block with a smaller force that helps to reduce the load fluctuation and deviation position from the passive compensator in that it outweighs the friction in the passive compensator. A control system regulates the force- and position load based on real time measurements from an acceleration sensor (MRU) on the vessel. In this way the performance of the total heave compensator system is improving in that all heave induced influences are eliminated. This is important when delicate operations are carried out on the seabed or in the well.

**[0003]** Publication US3946559 discloses an in-line compensator. These types of compensators are normally connected between the draw work and the drilling machine. The whole compensation system will therefore move accordingly when a lifting operation is performed by the derrick.

**[0004]** The compensation arrangement of the publication could be locked in a fully retracted position. The arrangement could in this position be used for round tripping, or the like, in the same manner as if the compensating arrangement were absent. However, the publication fails to disclose a completely disconnection of the heave compensator actuator from the compensation arrangement.

**[0005]** A further prior art arrangement is known from WO 2007/139394 A1 showing a compensation arrangement with a top mounted active heave compensation actuator.

**[0006]** In drilling rigs which are equipped with top-mounted active heave compensation actuator and passive heave compensator, the actuator of the active compensation is usually connected permanently to the movable part of the passive compensation machine.

**[0007]** The total amount of time the drilling rig is operational with active heave compensation is often limited compared with the time it is operational with passive heave compensation alone. The actuator of the active compensator is hence driven out and in by the spring force in the passive compensator in a greater portion of the time.

**[0008]** To minimize the abrasion and achieve optimal operation, it is advantageous to have an arrangement where the actuator of the active compensation can be connected and disconnected when it is needed. Such an arrangement will increase the lifetime of the active compensator actuator and increase the capacity of the passive compensator.

#### **Summary of the invention**

**[0009]** The invention provides an arrangement on a tower or derrick structure for connecting and disconnecting an active heave compensator actuator and/or additional passive compensator actuators which is operatively connected with a fixed structure at a first end. The arrangement comprises a connection device operatively connected to a crown block or other load carrying device, said connection device being adapted to selectively grip an adaptor head fixedly connected to a movable second end of the heave compensator actuator in order to hydraulically drive the second end of the heave compensator in and out of the first end of said heave compensator actuator, and that said arrangement further comprises a safety device attached to the tower or derrick structure and is adapted to selectively grip the adaptor head of the heave compensator actuator when said adaptor head is not in engagement with the connection device in order to hold the second end of the heave compensator actuator in a fixed position, said heave compensator actuator (1) being disconnected from the crown block or other load carrying device (4) when connected to the safety device (6).

**[0010]** Preferable embodiments of the device are defined in the dependent claims, to which reference is made.

**[0011]** The purpose of the device according to the present invention is to permit connection and disconnection of the active heave compensator actuator or additional passive compensator actuators against the movable crown block or other load carrying device which is passively compensated by fixed passive compensator actuators. The arrangement also protects against collisions between the movable end of the active compensator actuator and the movable crown block attached to the passive compensator. Usually the active compensator actuator is flexibly supported using a gimbal-mounting towards the tower structure. This is to

prevent transmission of lateral forces to the active compensator actuator when the movable crown-block is driven in and out, this may damage the seal assembly and the rod, because there always will occur lateral movement of the crown block. When the rod end on the active compensator actuator is disconnected from the crown block, the arrangement will need a support mechanism to prevent that the active compensator actuator tilt. In addition, the support arrangement contributes to a possible subsequent connection, in that the active compensator actuator is aligned to obtain the right entry (in this case vertical) of the connection device at the end of the actuator rod against the connecting arrangement on the typically passively compensated crown block.

**[0012]** The arrangement has the capability to connect and disconnect the dynamically loaded load bearing device, both when the actuator is set in motion or it is in stationary state.

## Figures

### [0013]

Fig. 1 shows a schematic view of an active compensator actuator with disconnection arrangement fitted and attached a passive compensator with movable crown block.

Fig. 2 shows a supporting arrangement 2 to support and secure the active compensator actuator,

1. a) shown in open position,
2. b) shown in closed position.

Fig 3 shows a safety device 6

1. a) shown in open position,
2. b) shown in closed position,
3. c) isometric view of the closed position.

Fig. 4 shows a connection arrangement 7 mounted on the movable passively compensated crown block for connection and disconnection of the active compensator actuator,

1. a) shown in open position,
2. b) shown in closed position,
3. c) isometric view of the closed position.

## Detailed description of the invention

**[0014]** Reference is made to Figure 1, illustrates the combination of connection and

disconnection arrangements 2, 6, 7, 8 arranged on a tower or a derrick structure 4 and in conjunction with the active compensator actuator 1 which are flexible supported via a gimbal 3 and connected to a movable crown block in a passive compensator 8. The arrangement comprises an actuated supporting device 2, a safety device 6 and a connection device 7. The safety device 6 and the connection device 7 being adapted to selectively grip an adaptor head 18.

**[0015]** The supporting device 3, the safety device 6, the connection device 7 and the connection with the adaptor head is further described in the following Figures 2-4.

**[0016]** Figure 2 shows the supporting arrangement 2.

The actuated supporting device 2 is mounted at or around the active compensator actuator 1. The supporting device 2 is connected to the derrick structure 4 with a hinged connection.

**[0017]** The supporting device 2 comprises three adjustable contact supports 9 that are run or placed against the stationary end of the active compensator actuator 1 using an arm construction. The arm construction comprising one main arm 10 and two linked arms 11, 13, positioned at an angle relative to the main arm 10. The main arm 10 and the linked arm 11 are hingedly connected at an upper end of the supporting device 2. The link 13 is connected to link arm 10 and supported in the lower end of device 2. The contact supports 9 are activated by hydraulically or electrically adjustable actuators 12.

**[0018]** The function of the supporting device 2 is to align the actuator when connecting and disconnecting as well as being a lateral support to the active compensator actuator 1 when this is disconnected. This is shown in Fig 2b. When the active compensator actuator 1 is connected to the crown block, the supporting arrangement 2 will be pulled away from the compensator actuator 1 and the contact supports 9 are not in contact with the compensator actuator 1. This is shown in Fig 2a.

**[0019]** Figure 3 shows the safety device 6. The safety device aims to ensure that the active compensator actuator rod 1 do not move when it is disconnected from the movable crown block in the passive compensator 8. The safety arrangement 6 comprises one or more gripping arms with parts 14-17 arranged to selectively grip a flange (not shown) on the adaptor head 18, the adaptor head 18 is spherical or has another shape and is connected to the movable part (the rod end) of the active compensator actuator 1. The gripping arms comprises an arm construction having a gripper 14 and two linked arms 15, 16 which are activated by one or more hydraulically or electrically adjustable actuators 17. The safety device comprises at least two arm constructions.

**[0020]** Figure 3a shows the safety device 6 in an open position where the grapplers 14 are not in contact with the adaptor head 18.

**[0021]** Figure 3b shows the safety arrangement in a closed position where the grapplers 14 engaging the flange of the adaptor head 18.

**[0022]** The safety device 6 is also attached to the derrick structure 4.

**[0023]** Figure 4 shows the connection device 7. The connection device 7 is connected to the movable and passively compensated crown block 8 in order to connect or disconnect the adaptor head 18 on the active compensated actuator 1, both when the passive compensated crown block 8 are in motion or in stationary condition. Using the connection device 7, the active compensator actuator 1 is connected to the crown block and will thus be able to be set in an active condition. The connection arrangement 7 is adapted to selectively grip a flange or spherical ball connector on the adaptor head 18, the adaptor head fixedly connected to the active compensator actuator 1 movable rod end (not shown). The connection device 7 comprises one or more gripping arms with parts 19-22 that are arranged to selectively grip the flange or spherical ball connector on the compensator actuator adaptor head 18. The arm construction comprising a grapples 19 and two link arms 20, 21. These are activated by hydraulically or electrically adjustable actuators 22.

**[0024]** Figure 4a shows the connection device 7 in an open position where the grapples 19 are not in contact with the adaptor head 18.

**[0025]** Figure 4b shows the safety connection device 7 in a closed position where the grapples 19 engaging the flange of the adaptor head 18.

**[0026]** When the adaptor head 18s connected to the connection device 7 the rod end (not shown) of the active compensator actuator 1 may be hydraulically driven in and out. When the adaptor head 18 is connected to the safety device 6, the rod end (not shown) of the active compensator actuator 1 is held in a fixed position.

**[0027]** The present invention has been described with reference to some preferred embodiments and some drawings for the sake of understanding only and it should be clear to persons skilled in the art that the present invention includes all legitimate modifications within the ambit of what has been described hereinbefore and claimed in the appended claims. Henceforward this connection and disconnection device can also be used for passive compensator actuators in passive compensators comprising two or more passive compensator actuators. This to allow using only the number of compensator actuators needed to achieve the necessary compensation force.

## **REFERENCES CITED IN THE DESCRIPTION**

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Patent documents cited in the description

- [US3946559A \[0003\]](#)
- [WO2007139394A1 \[0005\]](#)



## PATENTKRAV

1. Et arrangement (2, 6, 7, 8) på en tårn- eller boretårnskonstruktion (4) for tilslutning og frakobling af en aktiv og/eller passiv hævekompensatoraktuator (1), hvilken hævekompensatoraktuator (1) er operativt forbundet med en fast struktur i en første ende, **kendetegnet ved**, at arrangementet (2, 6, 7, 8) omfatter en forbindelsesindretning (7), der er operativt forbundet til en kronblok eller en anden lastbærende anordning (8), hvilken forbindelsesindretning (7) er tilpasset til selektivt at gribe fat i et adapterhoved (18), der er fast forbundet til en bevægelig anden ende af hævekompensatoraktuatoren (1) for hydraulisk at drive den anden ende af hævekompensatoraktuatoren (1) ind i og ud af den første ende af nævnte hævekompensatoraktuator (1), og at arrangementet (2, 6, 7, 8) yderligere omfatter en sikkerhedsindretning (6) fastgjort til tårn- eller boretårnskonstruktion (4) og er indrettet til selektivt at gribe fat i adapterhovedet (18) af hævekompensatoraktuatoren (1), når adapterhovedet (18) ikke er i indgreb med forbindelsesindretningen (7) for at holde den anden ende af hævekompensatoraktuatoren (1) i en fast position, hvor hævekompensatoraktuatoren (1) er frakoblet fra kroneblokken eller anden lastbærende anordning (8), når den er forbundet med sikkerhedsindretning (6).
2. Arrangement ifølge krav 1, **kendetegnet ved**, at forbindelsesindretningen (7) og sikkerhedsindretningen (6) er forsynet med gribearme (14-17,19, 22), som er indrettet til at gribe fat i adapterhovedet (18) af hævekompensatoraktuatorens (1) anden ende.
3. Arrangement ifølge krav 1 eller 2, **kendetegnet ved**, at arrangementet også omfatter en understøtningsindretning (2), som er indrettet til at understøtte hævekompensatoraktuatoren (1) lateralt, når den ikke er forbundet med kronblokken eller anden lastbærende anordning (8).
4. Arrangementet ifølge et hvilket som helst af de foregående krav, **kendetegnet ved**, at forbindelsesindretningen (7) og sikkerhedsindretningen (6) er indrettet til skiftevis at forbindes og frakobles under drift.
5. Arrangementet ifølge et hvilket som helst af de foregående krav, **kendetegnet ved**, at kronblokken er operativt forbundet med en passiv kompensator (8).
6. Arrangementet ifølge et hvilket som helst af de foregående krav, **kendetegnet ved**, at adapterhovedet (18) af den aktive hævekompensatoraktuator (1), der har et sfærisk adapterhoved, er indrettet til at blive grebet fat i af forbindelsesindretningen (7) og sikkerhedsindretningen (6).

# DRAWINGS

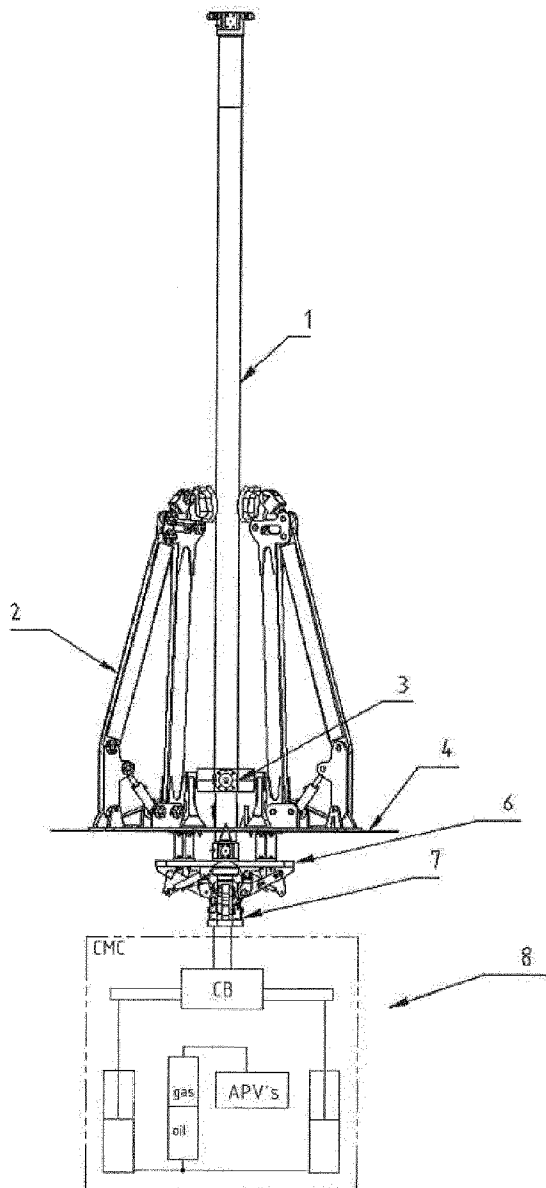


Fig. 1

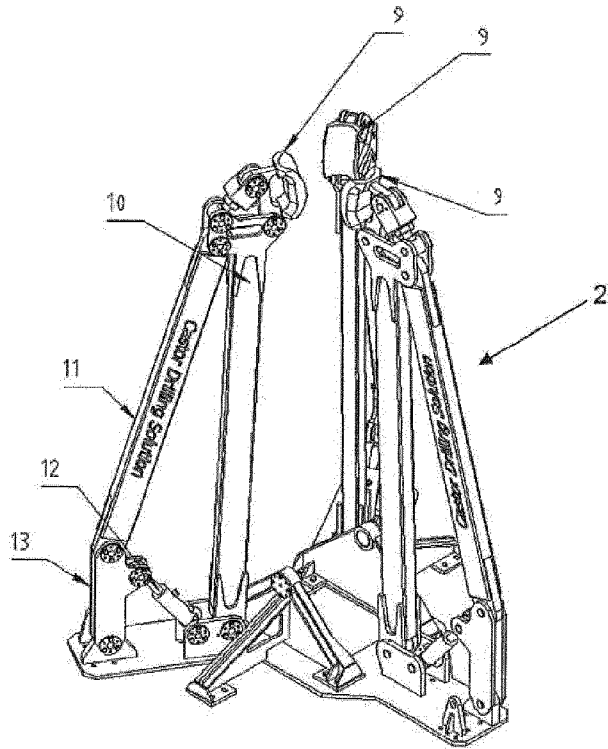


Fig. 2a

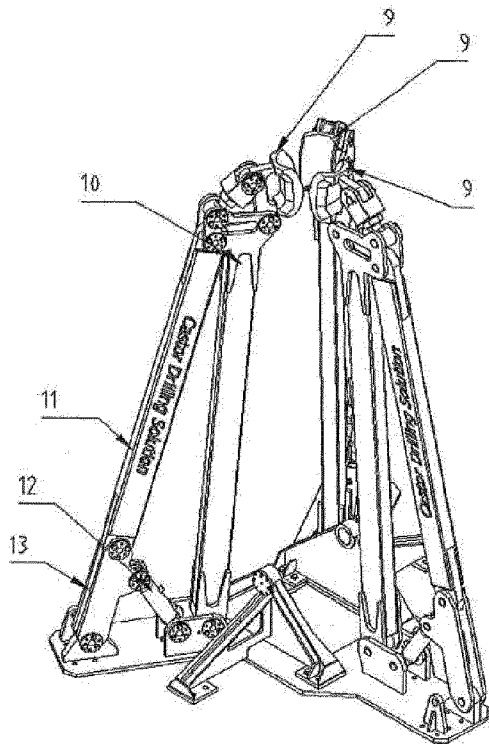


Fig. 2b

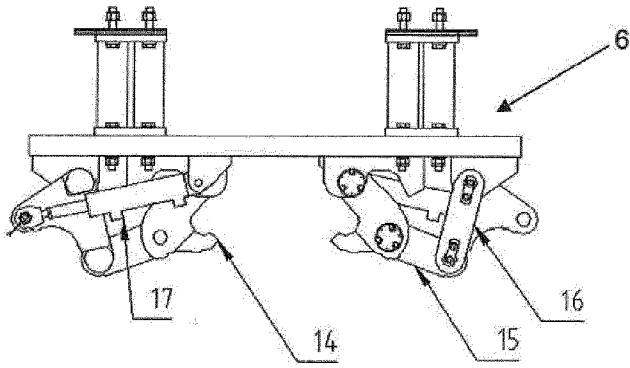


Fig. 3a

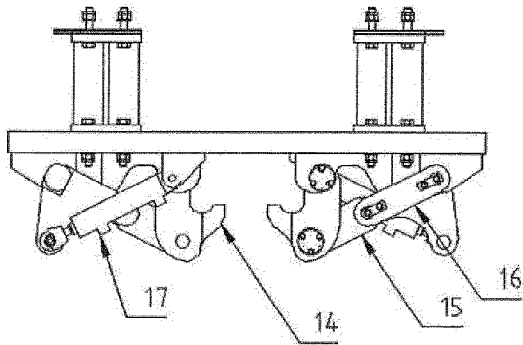


Fig. 3b

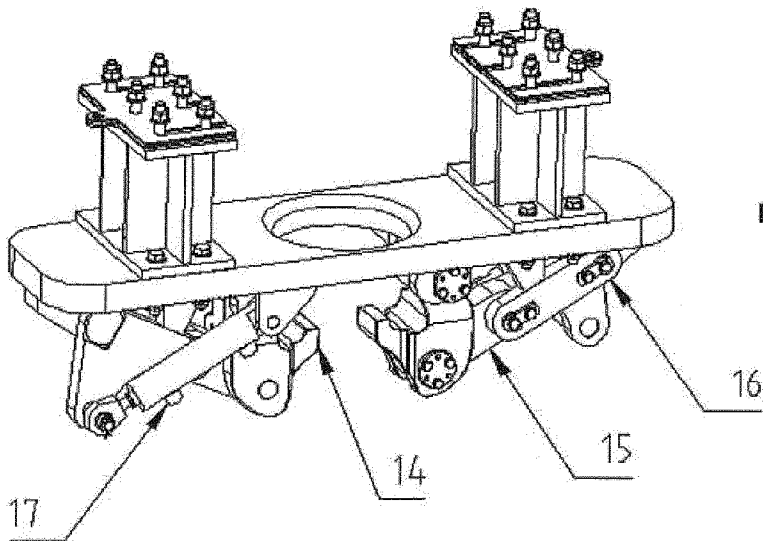


Fig. 3c

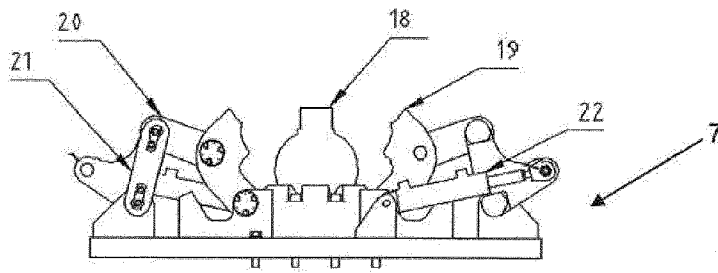


Fig. 4a

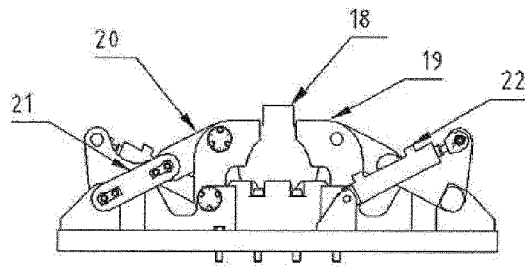


Fig. 4b

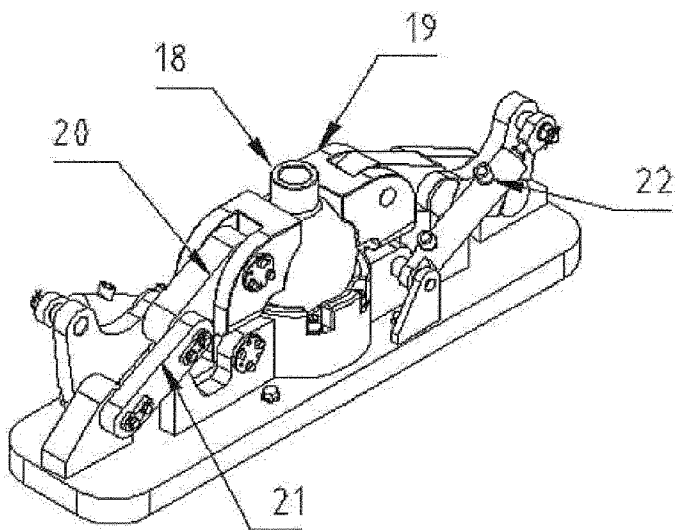


Fig. 4c