

March 15, 1949.

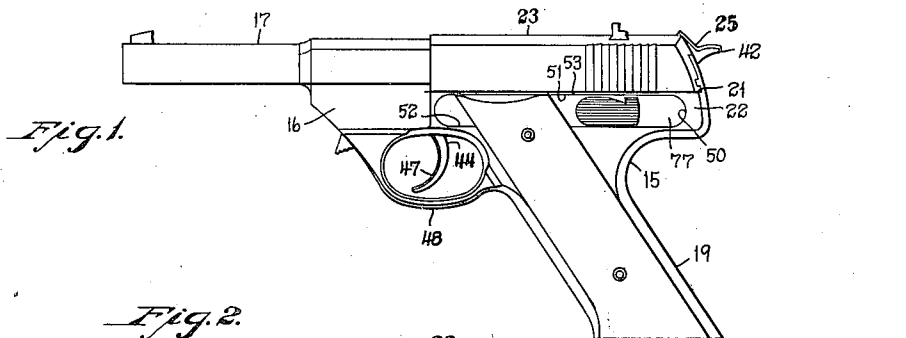
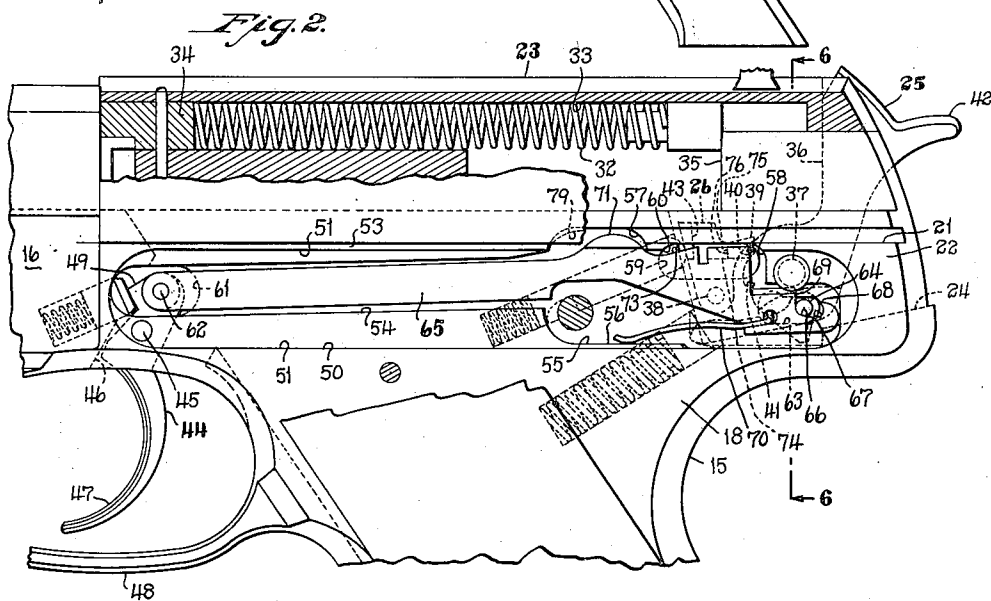
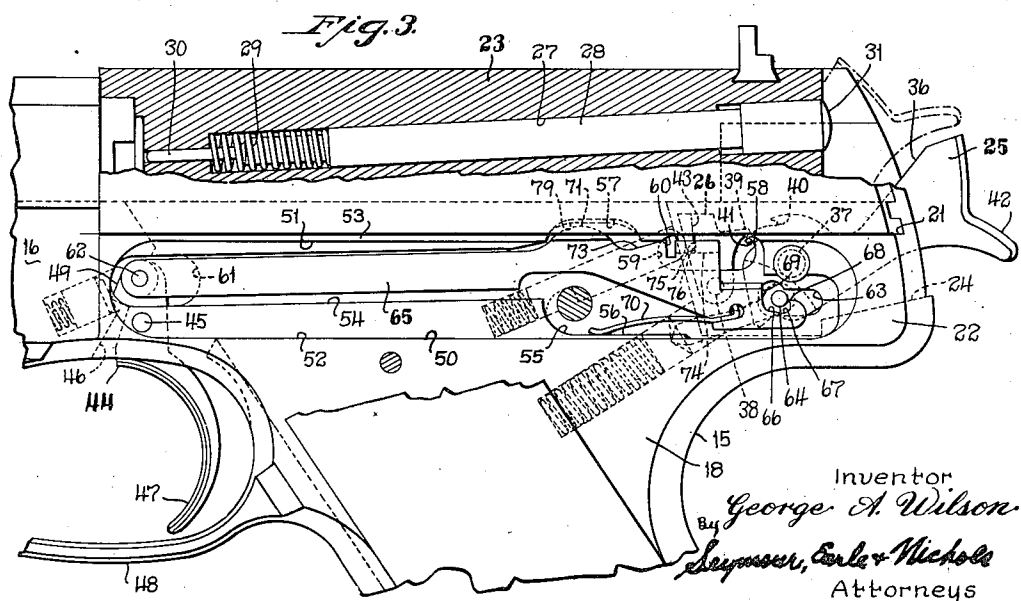
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2,464,427

DOUBLE-ACTION MECHANISM FOR PISTOLS

Filed Jan. 2, 1948

3 Sheets-Sheet 1



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3 Sheets-Sheet 2

Fig. 4.

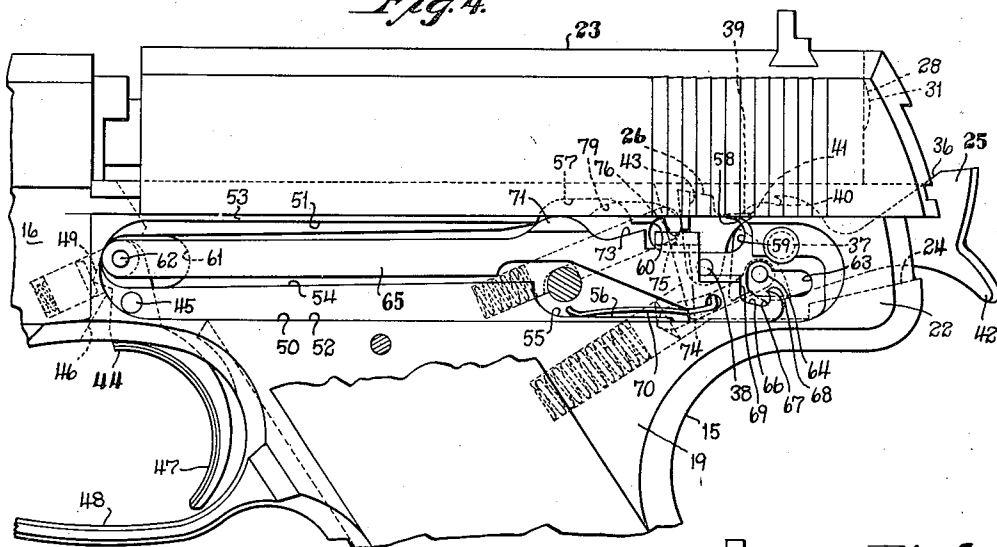


Fig. 5.

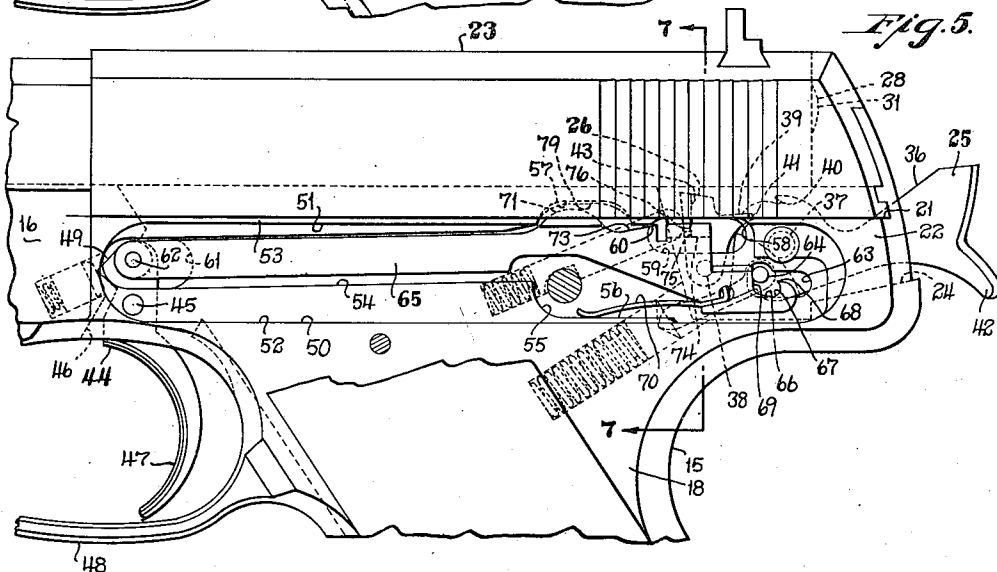
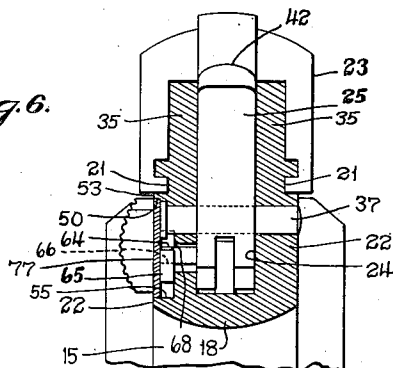


Fig. 6.



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3 Sheets-Sheet 3

Fig. 7.

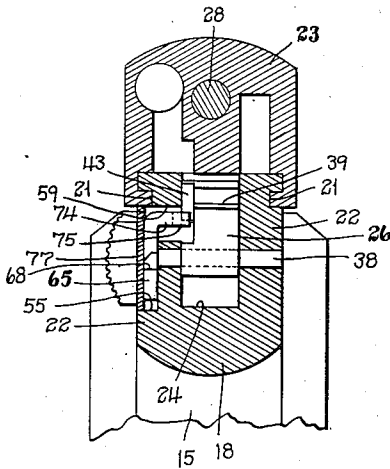


Fig. 8.

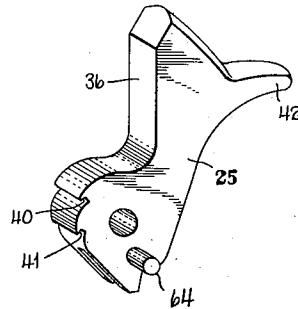


Fig. 11.

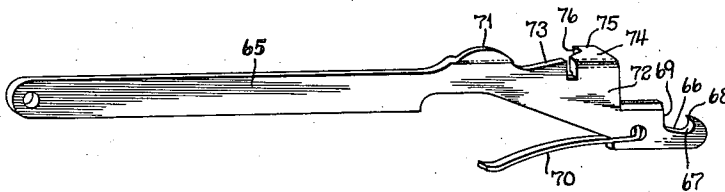


Fig. 10.

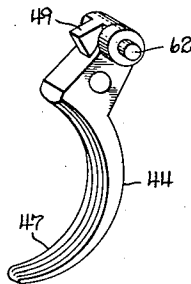
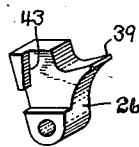


Fig. 9.



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2,464,427

DOUBLE-ACTION MECHANISM FOR PISTOLS

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Application January 2, 1948, Serial No. 65

5 Claims. (Cl. 89—195)

1

The present invention relates in general to fire-arms and more particularly to improvements in automatic pistols of the so-called "blow-back" type, wherein the breechblock is moved rearwardly relative to the barrel by gas pressure developed in the barrel and acting against the front face of the breechblock on discharge of a cartridge, the rearward excursion of the breechblock being utilized to extract the spent cartridge from the chamber of the barrel.

An object of the present invention is to provide a superior automatic blow-back type of pistol characterized by a double-action mechanism of economical and durable construction.

A further object of the invention is to provide an automatic blow-back type of pistol with a superior double-action mechanism embodying easily fabricated parts assembled in cooperative relationship with the fire-control mechanism of the pistol, so as to provide a direct and positive connection between the hammer and trigger.

A still further object of the invention is to provide an automatic blow-back type of pistol with superior double-action mechanism which renders the primary sear inoperative and itself performs the function of cocking and releasing the hammer on retraction of the trigger for firing the first round of ammunition.

With the above and other objects in view, as will appear to those skilled in the art from the present disclosure, this invention includes all features in the said disclosure which are novel over the prior art.

In the accompanying drawings, in which certain modes of carrying out the present invention are shown for illustrative purposes:

Fig. 1 is a side elevation of a pistol embodying the double-action mechanism of this invention;

Fig. 2 is an enlarged fragmentary side elevation with the cover-plate of the double-action mechanism removed and the hammer in its firing position;

Fig. 3 is similar to Fig. 2 but with the hammer held in cocked position by means of the double-action mechanism;

Fig. 4 is similar to Fig. 2 but with the hammer moved into cocking engagement with the primary sear by the action of the rearwardly moving breechblock;

Fig. 5 is similar to Fig. 4 but with the breechblock returned to its chamber-closing position and the trigger released;

Fig. 6 is an end elevation on section line 6—6 of Fig. 2;

2

Fig. 7 is an end elevation on section line 7—7 of Fig. 5;

Fig. 8 is a perspective view of the hammer;

Fig. 9 is a perspective view of the sear;

Fig. 10 is a perspective view of the trigger; and

Fig. 11 is a perspective view of the actuator and secondary sear-member of the double-action mechanism.

Referring to the drawings, the latter show an exemplary embodiment of the invention designed for operation in an automatic blow-back type of pistol which comprises, in the main, a frame 15 having a forward portion 16 on which the barrel 17 of the pistol is secured; and a rearward portion 18 provided with an integral downwardly-projecting hollow pistol-grip 19 arranged to receive a box-magazine, said rearward portion having longitudinal guide-grooves 21 in its opposite substantially-vertical sides 22 in which the breechblock 23 of the pistol is slidably supported for movement longitudinally relative to the fixed barrel. Intersecting the rear end of the rearward portion of the frame, is a vertical slot 24 which is adapted to accommodate the pivoted hammer 25 and primary sear 26 of the fire-control mechanism. Referring especially to Fig. 3, the breechblock has a longitudinal axial aperture 27 in which the firing-pin 28 is slidably mounted, the latter being urged normally rearwardly in the aperture 27 by a coil-spring 29 mounted on the forward end of the firing-pin and arranged normally to hold its cartridge-striking end 30 rearwardly of the front face of the breechblock and its hammer-engaging end 31 rearwardly of the rear end of the breechblock. The action-spring 32 of the piston illustrated herein is mounted in a longitudinal aperture 33 of the breechblock between a fixed abutment 34 in the forward end of the aperture 33, and one of a pair of fixed abutments 35 projecting upwardly from the rear end of the frame, the action-spring 32 being adapted thus to retard the rearward recoil of the breechblock following the discharge of the pistol and thereafter to force the breechblock forwardly into breech-closing position. During the rearward excursion of the breechblock, its rear end is adapted to engage the front face 36 of the hammer 25 above its transverse pivot-pin 37 to pivot the hammer rearwardly or clockwise, as seen in Fig. 4, against the resisting force of a conventional hammer-spring-and-link assembly into cocking engagement with the primary sear 26 of the pistol. The latter is pivoted as at 38 in the vertical slot 24 of the frame, forwardly of the hammer, its sear-nose 39 being urged into engagement

3

with either of two sear-notches 40 and 41 of the hammer by the force of a conventional sear-spring-and-pilot assembly so as to hold the hammer in its half- or full-cocked positions, respectively. Thus, the hammer is cocked automatically with the firing of each shot and the subsequent rearward excursion of the breech-block. In order to cock the hammer manually, the latter is provided with a knurled thumb-piece 42 by which the shooter may pull the hammer down manually to either its half- or full-cocked position.

Referring especially to Fig. 9, the sear has, in addition to its usual sear-nose 39, an integral shoulder 43 which projects laterally from and at substantially right angles to the vertical left-hand face of the sear, the shoulder being formed forwardly of the transverse pivot-pin 30 of the sear and sloped upwardly and forwardly at an angle of substantially 9° to a line perpendicular to the top edge thereof. The upper end of the sear-shoulder 43 intersects the top edge of the sear, while the lower end of the shoulder is squared off at a point approximately intermediate the top edge of the sear and the center of the sear-pin. This sear-shoulder 43 constitutes an element of the improved double-action mechanism of this invention, additional elements of which are described below.

Referring now to Figs. 2, 3 and 10, the trigger of the pistol is shown at 44 and is pivotally mounted on a transverse pivot-pin 45 in a vertical recess 46 in the forward part of the frame. The finger-piece 47 of the trigger projects downwardly into the trigger-guard 48 of the frame and is resiliently held forwardly in the trigger-guard by the force of a trigger-spring-and-pilot assembly acting against the upper end 49 of the trigger above its pivot-pin 45 to pivot the trigger clockwise, as seen in Fig. 2.

The double-action mechanism is adapted to connect the trigger to the hammer of the pistol so that the first shot may be fired simply by pulling the trigger, the rearward draft on the trigger serving both to cock the hammer and thereafter to release the latter against the firing-pin. The double-action mechanism serves also in the normal operation of the pistol, that is to say, when the hammer is cocked by the primary sear, to disengage the latter therefrom and fire the pistol. The double-action mechanism is mounted, preferably, in one of the vertical sides 22 of the pistol-frame and to this end, the left-hand side of the latter is provided with a longitudinal relatively-shallow recess 50, the over-all length of which is slightly greater than the distance between the pivot-pins of the trigger and hammer, respectively. As shown, the opposite ends of the recess 50 are preferably arcuate and the upper and lower sides 51 and 52 respectively of the recess are substantially parallel, the lower side extending longitudinally of the rear end-portion of the frame immediately above the juncture of the upper end of the pistol-grip 19 therewith; and the upper side 51 extending parallel to and immediately below the corresponding guide-groove 21 of the frame, being separated therefrom by a longitudinal rib 53. Formed in the bottom of the shallow recess 50 is a longitudinal groove 54 which extends from the forward end of the recess rearwardly and upwardly at a relatively slight angle to a point substantially intermediate the opposite ends of the recess, at which point the longitudinal groove 54 intersects a cutout 55 in the bottom of the

4

recess 50, this cutout being substantially equal in depth to that of the groove 54 and extending substantially to the rear end of the recess.

The forward edge of the bottom half of the cutout forms a ledge 56 with the lower edge of the recess 50, while the top edge of the cutout immediately rearwardly of the rear end of the longitudinal groove 54 is extended upwardly to intersect the longitudinal guide-groove 21 of the frame, the intersecting portion of the cutout being elongated and hereinafter referred to as the clearance-recess 57. The top edge of the cutout also intersects the separating-rib 53 rearwardly of the clearance-recess 57 to form an elongated clearance-slot 58, which constitutes the upper edge of an elongated aperture 59. The latter extends inwardly transversely to intersect the vertical hammer-and-sear recess 24 at the rear end of the frame substantially opposite the shoulder 43 of the sear to provide access thereto from the cutout 55. Between the forward end of the clearance-slot 58 and the clearance-recess 57 is a relatively-short length of the separating-rib 53, the bottom rear corner of which constitutes a cam-engaging surface 60 for the purpose hereinafter described. Formed in the forward end of the longitudinal groove 54 is an elongated aperture 61 which extends transversely of the frame and is adapted to intersect the upper end of its vertical trigger-recess 46 to provide access to the upper end of the trigger from the groove in the left-hand side of the frame. As shown especially well in Fig. 10, a shouldered trigger-pin 62 is secured to the upper end of the trigger substantially opposite the elongated aperture 61 so as to extend outwardly transversely therethrough into the longitudinal groove 54 of the frame. A second elongated aperture 63 is provided in the cutout 55 adjacent the rear end thereof and directly beneath the pivot-pin 37 of the hammer, this aperture 63 extending transversely into the vertical hammer-and-sear slot 24 of the frame to provide access to the hammer from the cutout in the left-hand side of the frame. As shown especially well in Fig. 8, a pin 64 is carried on the hammer below and slightly to the rear of its pivot-pin 37, and is adapted to extend outwardly transversely through the elongated aperture 63 into the rear end of the cutout 55. Referring especially to Figs. 2, 5 and 11, the double-action mechanism comprises, in the main, a link-member 65 which is slidably mounted in the longitudinal groove 54 and is pivotally connected at its forward end to the outer end of the shouldered trigger-pin 62 which extends outwardly through the elongated aperture 61 in the forward end of the groove 54. The width of the link-member is somewhat less than the width of the longitudinal groove 54 so as to permit the link-member to move freely upwardly and forwardly therein on retraction of the finger-piece of the trigger. The rear end of the link-member is normally at substantially the rear end of the cutout and is characterized by a longitudinally-elongated notch 66 in its upper edge, the rear wall of which constitutes a radius curving upwardly and forwardly, the lower portion thereof constituting a cocking-abutment 67 and the upper portion thereof comprising a secondary sear-shoulder 68, for the purpose hereinafter described.

The forward wall of the notch 66 constitutes a substantially-vertical rearwardly-facing shoulder 69 hereinafter referred to as a thrust abutment 75 against which the transversely-projecting pin of

5

6

the hammer is adapted to engage for thrusting the link-member forwardly in its groove 54, as the hammer is pivoted rearwardly into its half-cocked or full-cocked positions. When the hammer is in its normal uncocked position, the thrust abutment 69 is held against the transverse pin 64 of the hammer by the force of the trigger-spring acting normally to thrust the link-member rearwardly. The elongated notch 66 of the link-member is adapted to be held up normally in cooperative relationship with the transverse pin 64 of the hammer by the force of a resilient-member 70 comprising a leaf-spring frictionally secured at its rear end in a slot in the underside and adjacent the rear end of the link-member; and slidably engageable at its forward end with the stepped ledge 56 of the cutout. Formed integrally on the upper edge of the link-member and immediately opposite the clearance-recess 57 of the cutout 55 is a disconnecter-means 71 comprising an upwardly-projecting lug rounded over on its opposite ends and adapted to project upwardly and outwardly into the guide-groove 21 of the frame and to move longitudinally freely in the elongated clearance-recess 57 of the cutout 55. The portion of the link-member between its disconnecter-means 71 and its elongated notch 66 slopes downwardly and rearwardly and is provided on its upper edge with an integral upwardly-projecting flange-portion 72, the forward edge of which comprises an upwardly-and-rearwardly-sloping cam-surface 73 arranged normally beneath and rearwardly of the cam-engaging surface 60 of the separating-rib 53 and adapted to cooperatively engage therewith in the manner and for the purpose hereinafter described. Rearwardly of the cam-surface 73 of the flange 72, the latter is provided with an integral lip-portion 74 which is arranged to extend inwardly freely into the elongated transverse aperture 59 of the cutout, the forward edge of the lip 74 sloping rearwardly and having a substantially-rectangular tongue-portion 75 at its inner end, the forward edge 76 of which is located normally rearwardly of the shoulder 43 of the sear and is adapted to engage the latter and move it out of operative engagement with the hammer as hereinafter described, the forward edge 76 of the tongue being hereinafter referred to as a "primary sear-abutment." The recess 50 of the frame is adapted to accommodate a cover 77 (see Fig. 1) which not only protects the operating parts of the double-action mechanism from dirt, moisture, and the like, but also serves to hold the link-member of the double-action mechanism in assembled position on the respective pins of the trigger and hammer, the cover being fixedly secured in the recess 50 in any suitable manner.

The operation of the double-action mechanism may be described briefly as follows:

On retracting the finger-piece 47 of the trigger, the link-member 65 is pulled forwardly in its groove 54 and cutout 55 of the frame, thereby bringing the cocking-abutment 67 of the elongated notch 66 at the rear end of the link-member into engagement with the back side of the transversely extending hammer-pin 64. Continued rearward movement of the finger-piece 47 of the trigger pulls the hammer-pin forwardly, thereby pivoting the hammer 25 clockwise about its transverse pivot-pin 37 against the resisting force of the hammer-spring and advancing the cam-surface 73 of the link-member into juxtaposition to the cam-engaging surface 60 of the separating-rib 53. On continuing the forward move-

ment of the link-member, the primary sear-abutment 76 thereof engages the shoulder 43 of the primary sear and pivots the latter counterclockwise, thereby withdrawing its sear-nose 39 out of cooperative relationship with the sear-notches of the hammer. Further rearward movement of the trigger and the consequent forward movement of the link-member then brings the cam-surface 73 of the link-member into engagement with the cam-engaging surface 60 of the separating-rib of the frame, whereupon the rear end of the link-member is depressed against the restraining force of the leaf-spring 70. With the depression of the rear end of the link-member, its cocking-abutment 67 is withdrawn from engagement with the back side of the transverse pin 64 of the hammer, the secondary sear-shoulder 68 of the notch initially engaging and almost immediately thereafter disengaging the hammer-pin 64, thereby releasing the hammer which has been pivoted nearly into its full-cocked position. The compressed hammer-spring thereupon swings the hammer forwardly forcefully against the rear end of the firing-pin to discharge a cartridge in the chamber of the barrel. Immediately on discharge of the cartridge, the breechblock 23 begins to recede from the fixed barrel sliding rearwardly in the guide-grooves 21 of the frame. As the breechblock begins its rearward excursion, a clearance-recess 79 provided in the left-hand side thereof for accommodating the disconnecter-lug 71 of the link-member, is displaced rearwardly relative to the disconnecter-lug 71, the forward edge of the clearance-recess 79 cammingly engaging the rounded front edge of the lug 71, thereby forcing the latter downwardly, whereupon the primary sear-abutment 76 of the link-member is withdrawn from engagement with the shoulder 43 of the primary sear so as to release the latter and permit it to return into cooperative relationship with the hammer. Consequently, as the latter is forced back automatically into its cocked position by the rearwardly-moving breechblock, the primary sear will initially ride over and subsequently engage the sear-notch 41 of the hammer to hold the latter in its full-cocked position, following the return of the breechblock to its forward breech-closing position. With the movement of the sear into the position for cocking engagement with the hammer, the lower end of the sear-shoulder 43 of the primary sear 26 rides over the tongue 75 of the depressed link-member to hold the rear end of the latter down, as a consequence of which it is impossible to disengage the sear from the hammer without first releasing the trigger so as to permit the link-member and more particularly its tongue to move rearwardly out from beneath the lower end of the sear-shoulder. On releasing the trigger, the force of the trigger-spring thrusts the link-member rearwardly and simultaneously the leaf-spring urges the rear end of the link-member upwardly, whereby the tongue 75 of the link-member moves rearwardly and upwardly from beneath the lower end of the sear-shoulder 43 to position the primary sear-abutment 76 in juxtaposition to the rear face thereof. It will be appreciated that inasmuch as the leaf-spring of the link-member urges the rear end thereof upwardly, the thrust-abutment of its elongated notch engages against the transverse pin of the hammer, and that since the latter is in its full-cocked position, its pin is in its foremost position in its aperture 63, whereby the link-member is held forwardly of its normal rear-most position, that is to say, the trigger remains

in a partially-retracted position (see Fig. 5) in which the primary sear-abutment 76 of the tongue 75 is immediately rearwardly of the shoulder of the primary sear. Consequently, a relatively-short rearward pull on the finger-piece of the trigger will move the primary sear-abutment 76 of the link-member forwardly into engagement with the primary sear and disengage the latter from the sear-shoulder 41 of the hammer to permit it to swing forwardly for firing the next-succeeding shot. After the final shot has been fired and the trigger pulled once more to release the hammer from its cocked position, the latter returns to firing position wherein its transverse pin 64 is at its rearmost position in its aperture 63, and hence on releasing the trigger, the link-member and more particularly the thrust-abutment 69 of its elongated notch, moves rearwardly in the cutout to its original rearmost position therein, the leaf-spring of the link-member elevating the rear end of the link upwardly so as to engage the cocking-abutment 67 of the elongated notch with the rear face of the hammer-pin 65. The arm is thus readied for firing the first round of a new magazine on retracting the trigger.

As pointed out above, the double-action mechanism comprises essentially a single link-member which functions as an actuator of the hammer and primary sear of the firearm and also as a secondary sear to release the hammer. Moreover, the double-action mechanism embodies a disconnecter operable by the relative movement of the breechlock to prevent doubling, the link-member of the double-action mechanism being characterized by its simplicity of construction and operation, the entire assembly being retained within the groove in the side of the frame and effectively concealed and retained in operative relationship therein by the cover-member fitting into the recess in the side of the frame.

The invention may be carried out in other specific ways than those herein set forth without departing from the spirit and essential characteristics of the invention, and the present embodiments are, therefore, to be considered in all respects as illustrative and not restrictive, and all changes coming within the meaning and equivalency range of the appended claims are intended to be embraced therein.

I claim:

1. An automatic pistol of the blow-back type, including in combination: a frame; fire-control mechanism including a trigger mounted in the forward part of said frame, a hammer mounted in the rear part thereof, and a primary sear having a sear-shoulder; and double-action mechanism to connect said trigger and said hammer, said mechanism comprising an actuator-and-secondary-sear-member arranged to be moved forwardly by retraction of said trigger and having a cocking-abutment adjacent its rear end and a thrust-abutment forwardly thereof, said cocking-abutment being arranged normally to engage said hammer and sequentially to cock said hammer and then to disengage the latter as a result of the forward movement of said actuator-and-secondary-sear-member by retraction of said trigger, a spring carried by said actuator-and-secondary-sear-member and arranged to cooperatively engage said frame to urge said cocking-abutment into re-engagement with said hammer on release of said trigger, a primary abutment arranged to be actuated by the forward movement of said actuator-and-secondary-sear-member to engage said sear-shoulder and disable said pri-

mary sear prior to the release of said hammer, and a disconnecter operable on release of said hammer and the subsequent discharge of said pistol to positively withdraw said primary sear-abutment from disabling engagement with the sear-shoulder of said primary sear so as to permit said primary sear to cock said hammer, said actuator-and-secondary-sear-member being arranged also to be moved forwardly by engagement of said hammer with the thrust-abutment thereof when said hammer is moved into cocked engagement with said primary sear to position said primary sear-abutment in cooperative relationship with said sear-shoulder, thereby to disengage said primary sear from said hammer on retraction of said trigger.

2. An automatic pistol of the blow-back type, including in combination: a frame having a cam-surface; fire-control mechanism including a trigger mounted in the forward part of said frame, a hammer mounted in the rear part thereof, and a primary sear; and double-action mechanism to connect said trigger and said hammer, said mechanism comprising an actuator-and-secondary-sear-member arranged to be moved forwardly by retraction of said trigger and having a cam-surface arranged to cooperatively engage with the cam-surface of said frame during the forward movement of said actuator-and-secondary-sear-member to depress the rear end thereof, a cocking-abutment at the rear end of said actuator-and-secondary-sear-member and a thrust-abutment forwardly thereof, said cocking-abutment being arranged normally to engage said hammer and sequentially to cock said hammer as a result of the forward movement of said actuator-and-secondary-sear-member by retraction of said trigger and thereafter to disengage said hammer as a result of the depression of the rear end of said actuator-and-secondary-sear-member, and a primary sear-abutment arranged to be actuated by the forward movement of said actuator-and-secondary-sear-member to engage and disable said primary sear prior to the release of said hammer, said actuator-and-secondary-sear-member being arranged also to be moved forwardly by engagement of said hammer with the thrust-abutment thereof when said hammer is moved into cocked engagement with said primary sear to position said primary sear-abutment in cooperative relationship thereto, thereby to disengage said primary sear from said hammer on retraction of said trigger.

3. An automatic pistol of the blow-back type, including in combination: a frame having a cam-surface; fire-control mechanism including a trigger mounted in the forward part of said frame, a hammer mounted in the rear part thereof, and a primary sear; and double-action mechanism to connect said trigger and said hammer, said mechanism comprising an actuator-and-secondary-sear-member arranged to be moved forwardly by retraction of said trigger and having a cam-surface arranged to cooperatively engage with the cam-surface of said frame during the forward movement of said actuator-and-secondary-sear-member to depress the rear end thereof, a cocking-abutment at the rear end of said actuator-and-secondary-sear-member and a thrust-abutment forwardly thereof, said cocking-abutment being arranged normally to engage said hammer and sequentially to cock said hammer as a result of the forward movement of said actuator-and-secondary-sear-member by retraction of said trigger and thereafter to disengage

said hammer as a result of the depression of the rear end of said actuator-and-secondary-sear-member, a leaf-spring secured at its rear end to said actuator-and-secondary-sear-member and slidably engaged at its forward end with said frame to urge said cocking-abutment into re-engagement with said hammer on release of said trigger, and a primary sear-abutment arranged to be actuated by the forward movement of said actuator-and-secondary-sear-member to engage and disable said primary sear prior to the release of said hammer, said actuator-and-secondary-sear-member being arranged also to be moved forwardly by engagement of said hammer with the thrust-abutment thereof when said hammer is moved into cocked engagement with said primary sear to position said primary sear-abutment in cooperative relationship thereto, thereby to disengage said primary sear from said hammer on retraction of said trigger.

4. An automatic pistol of the blow-back type, including in combination: a frame having a cam-surface; a breechblock reciprocally mounted on said frame; fire-control mechanism including a trigger mounted in the forward part of said frame, a hammer mounted in the rear part thereof, and a primary sear; and double-action mechanism to connect said trigger and said hammer, said mechanism comprising an actuator-and-secondary-sear-member arranged to be moved forwardly by retraction of said trigger and having a cam-surface arranged to be moved into cooperative engagement with the cam-surface of said frame during the forward movement of said actuator-and-secondary-sear-member to depress the rear end thereof, a cocking-abutment at the rear end of said actuator-and-secondary-sear-member and a thrust-abutment forwardly thereof, said cocking-abutment being arranged normally to engage said hammer and sequentially to cock said hammer as a result of the forward movement of said actuator-and-secondary-sear-member by retraction of said trigger and then to disengage said hammer as a result of the depression of the rear end of said actuator-and-secondary-sear-member, a primary sear-abutment arranged to be actuated by the forward movement of said actuator-and-secondary-sear-member to engage and disable said primary sear prior to the release of said hammer and a disconnecter carried by said actuator-and-secondary-sear-member arranged to be cooperatively engaged by said breechblock following the discharge of said pistol to positively withdraw said primary sear-abutment from disabling engagement with said primary sear so as to permit said primary sear to cock said hammer, said actuator-and-secondary-sear-member being arranged also to be moved forwardly by engagement of said hammer with the thrust-abutment thereof when said hammer is moved into cocked

engagement with said primary sear to position said primary sear-abutment in cooperative relationship thereto, thereby to disengage said primary sear from said hammer on retraction of said trigger.

5. An automatic pistol of the blow-back type, including in combination; a frame having a cam-surface; a breechblock reciprocally mounted on said frame; fire-control mechanism including a trigger mounted in the forward part of said frame, a hammer mounted in the rear part thereof, and a primary sear having a sear-shoulder; and double-action mechanism to connect said trigger and said hammer, said mechanism comprising an actuator-and-secondary-sear-member arranged to be moved forwardly by retraction of said trigger and having a cam-surface arranged to be moved into cooperative engagement with the cam-surface of said frame during the forward movement of said actuator-and-secondary-sear-member to depress the rear end thereof, a cocking-abutment at the rear end of said actuator-and-secondary-sear-member and a thrust-abutment forwardly thereof, said cocking-abutment being arranged normally to engage said hammer and sequentially to cock said hammer as a result of the forward movement of said actuator-and-secondary-sear-member by retraction of said trigger and then to disengage said hammer as a result of the depression of the rear end of said actuator-and-secondary-sear-member, a leaf-spring secured at its rear end to said actuator-and-secondary-sear-member and slidably engageable at its forward end with said frame to urge said cocking-abutment into re-engagement with said hammer on release of said trigger, a primary sear-abutment arranged to be actuated by the forward movement of said actuator-and-secondary-sear-member to engage said sear-shoulder and disable said primary sear prior to the release of said hammer, and a disconnecter carried by said actuator-and-secondary-sear-member arranged to be cooperatively engaged by said breechblock following the discharge of said pistol to positively withdraw said primary sear-abutment from disabling engagement with the sear-shoulder of said primary sear so as to permit said primary sear to cock said hammer, said actuator-and-secondary-sear-member being arranged also to be moved forwardly by engagement of said hammer with the thrust-abutment thereof when said hammer is moved into cocked engagement with said primary sear to position said primary sear-abutment in cooperative relationship with said sear-shoulder, thereby to disengage said primary sear from said hammer on retraction of said trigger.

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