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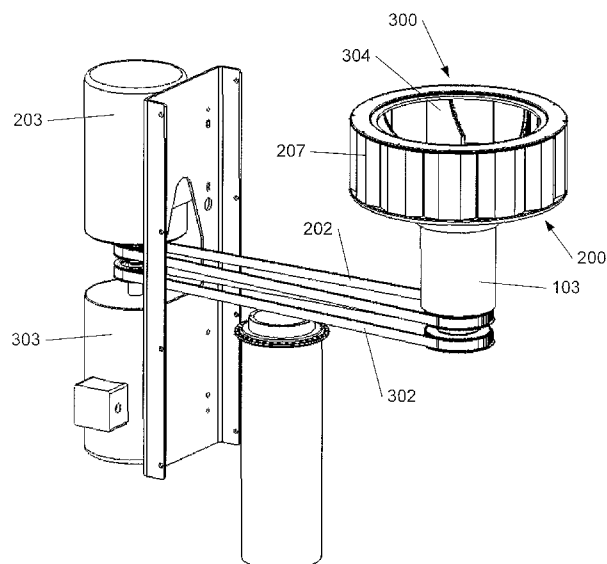


Fig. 5

(57) Abstract: Apparatus for cutting products, comprising: a base (100); a cutting head (200) with at least one cutting element along the circumference of the cutting head for cutting products fed into the cutting head, the cutting head being rotatably fitted to the base; an impeller (300) adapted for rotating concentrically within the cutting head (200) to urge products fed into the cutting head towards the circumference of the cutting head by means of centrifugal force; a first drive mechanism (301-303) for driving the rotation of the impeller at a first rotational speed setting the centrifugal force; and a second drive mechanism (201-203) for driving the rotation of the cutting head at a second rotational speed, determined such with respect to the first rotational speed that the product is cut by the at least one cutting element at a predetermined cutting velocity



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Apparatus and method for cutting products

Technical field

The present invention relates to an apparatus for cutting products, such as for example food products or ingredients for pharmaceuticals or the like, comprising an impeller which can rotate concentrically within a cutting head to impart centrifugal force to the products to be cut.

The present invention further relates to a method for cutting a product in which the product is fed to a cutting head in which an impeller rotates concentrically to impart centrifugal force to the product.

Background art

An apparatus for cutting food products of the type comprising an impeller rotating inside a cutting head is known for example from US-A-6968765. The cutting head is a stationary drum which is fitted with multiple cutting stations. Products cut with this technology include potato chips, cheese shreds, vegetable slicing, nut slicing and countless others. Centrifugal force is required to apply pressure to the product for stability when it passes the blades in the cutting stations. The centrifugal force is specific to the product, but it is known that too high centrifugal force can produce excess friction and compression on the product and that too low centrifugal force can cause poor knife engagement resulting in damage of the product. The desired cutting velocity is also specific for a given product.

In this type of apparatus, the cutting velocity is directly related to centrifugal force as both depend directly on the rotational speed of the impeller. However, the optimal impeller rotational speed from a viewpoint of centrifugal force is often different from the optimal impeller rotational speed from a viewpoint of cutting velocity. In those cases, upon selecting the impeller rotational speed a trade-off has to be made between more optimal centrifugal force and more optimal cutting velocity.

Disclosure of the invention

It is an aim of the present invention to provide an improved apparatus for cutting products of the type comprising an impeller rotating
5 inside a cutting head.

It is another aim of the present invention to provide an improved method for cutting products by means of a cutting head in which an impeller rotates.

These and other aims are achieved according to the invention as
10 defined in the claims.

As used herein, "rotational speed" is intended to mean the speed at which an object rotates around a given axis, i.e. how many rotations the object completes per time unit. A synonym of rotational speed is speed of revolution. Rotational speed is commonly expressed in RPM (revolutions
15 per minute).

As used herein, "cutting velocity" is intended to mean the speed at which a cutting element cuts through a product or alternatively states the speed at which a product passes a cutting element. Cutting velocity is commonly expressed in m/sec.

As used herein, a "cutting element" is intended to mean any element which is configured for cutting a particle or a piece from an object or otherwise reducing the size of the object, such as for example a knife, a blade, a grating surface, a cutting edge, a milling element, a comminuting element, a cutting element having multiple blades, etc., the foregoing
20 being non-limiting examples.

According to an aspect of the invention, which may be combined with other aspects described herein, the impeller is rotated by means of a first drive mechanism at a first rotational speed, which sets the centrifugal force imparted to the product. The cutting head is no longer stationary as
25 in the prior art document US-A-6968765 but can be rotated by means of a second drive mechanism at a second rotational speed. The second

rotational speed is determined such with respect to the first rotational speed that the product is cut by the at least one cutting element at a predetermined cutting velocity. By determining the second rotational speed in relation to the first rotational speed, the cutting velocity is set. For
5 example, if the cutting head and the impeller rotate in the same direction, the cutting velocity is proportional to the first rotational speed minus the second rotational speed. For example, if the cutting head and the impeller rotate in opposite directions, the cutting velocity is proportional to the sum of the absolute values of the rotation speeds.

10 According to this aspect, the centrifugal force and the cutting velocity can be made independent from each other. The centrifugal force is still proportional to the first rotational speed of the impeller like in the prior art, but the cutting velocity is now dependent on the first rotational speed of the impeller and the second rotational speed of the cutting head.
15 As a result, by establishing the first and second rotational speeds, both the centrifugal force and the cutting velocity can be optimized for the product which is to be cut and the need for making a trade-off like in the prior art can be avoided.

According to an aspect of the invention, which may be combined
20 with other aspects described herein, the first and second drive mechanisms are provided with controls for adjusting the first and second rotational speeds within respectively a first range and a second range. In this way, the cutting velocity and the centrifugal force can be established for a wide range of products. The controls can comprise a user interface,
25 by means of which the user can set the first and second rotational speeds. The controls can also be adjusted by means of another device, such as for example a PLC which takes a feedback input from sensors which sense for example temperature, product density, or other parameters, and on the basis thereof adjusts the rotational speeds. Another example is the use of
30 the apparatus for cutting potato chips in combination with a fryer for frying the potato chips. In this case the controls can be adjusted on the basis of

fryer requirements. One such requirement is for example a supply of
 potato chips to the fryer which is as uniform as possible, which means that
 the cutting apparatus has to be speeded up or slowed down to a given
 extent at times. Up to now, this speeding up or slowing down could lead to
 5 a significant amount of miscuts and product damage. With the apparatus
 of the invention, this can be minimised, as the centrifugal force can be
 optimised.

According to an aspect of the invention, which may be combined
 with other aspects described herein, the first drive mechanism comprises a
 10 first drive shaft by which the impeller is driven and the second drive
 mechanism comprises a second drive shaft by which the cutting head is
 driven, the second drive shaft being hollow and the first drive shaft being
 rotatably mounted within the second drive shaft. This has the advantage
 that the impeller and the cutting head are driven from the same side, e.g.
 15 the bottom side, leaving the top side unobstructed for feeding the product
 into the cutting head.

According to an aspect of the invention, which may be combined
 with other aspects described herein, the first and second drive
 mechanisms can have separate motors, so that the rotation of the impeller
 20 is entirely independent from the rotation of the cutting head. This has the
 advantage that the cutting velocity is totally independent of the centrifugal
 force.

In preferred embodiments wherein the apparatus has separate
 motors, the impeller is directly driven by the first motor of the first drive
 25 mechanism and the cutting head is directly driven by the second motor of
 the second drive mechanism. This has the advantages that any
 intermediate drive components can be avoided and the construction can
 be simplified. Preferably, in such embodiments, the base comprises a post
 with a first arm carrying the first motor with the impeller and a second arm
 30 carrying the second motor with the cutting head, the second arm being
 movably mounted to the post in such a way that the cutting head can be

removed from around the impeller. Preferably, in such embodiments, the rotation of the impeller inside the cutting head is stabilised by means of a spring-loaded pin on the impeller which fits into a tapered hole in the centre of the cutting head, or vice versa.

5 In other embodiments, the first and second drive mechanisms can have a shared motor, which drives the rotation of both the impeller and the cutting head, and a gearbox, by means of which the difference between the first rotational speed of the impeller and the second rotational speed of the cutting head can be set. The gearbox can have multiple gears, so that
10 different ratios between the first and second rotational speeds can be set.

 In preferred embodiments, the cutting head and the impeller can be oriented to rotate around a vertical axis or a horizontal axis. However, other angles with respect to horizontal are also possible.

 In preferred embodiments, the cutting head and the impeller are
15 mounted on a tiltable part of the base, by means of which the rotation axis of the cutting head and the impeller can be tilted to different angles. In this way, the orientation of the rotation axis can be adapted.

 According to an aspect of the invention, which may be combined with other aspects described herein, the cutting head comprises a
20 releasable locking mechanism for releasably fixing the cutting head to the base without using tools.

 According to an aspect of the invention, which may be combined with other aspects described herein, the cutting head can be made
25 stationary if desired, for example for use in conjunction with a dicing unit which is mounted at the outside of the cutting head.

Brief description of the drawings

 The invention will be further elucidated by means of the following description and the appended figures.

30 Figure 1 shows a perspective view of an impeller of a prior art cutting apparatus.

Figure 2 shows a perspective view of a cutting head of a prior art cutting apparatus.

Figure 3 shows a cross sectional perspective view of the impeller and cutting head of the prior art apparatus, mounted inside each other.

5 Figure 4 shows a perspective view of a first preferred embodiment of a cutting apparatus according to the invention.

Figure 5 shows a perspective view of the first embodiment of figure 4 with some parts removed in order to show its operation.

10 Figure 6 shows a perspective view of the impeller of the first embodiment of figure 4.

Figure 7 shows a perspective view of the cutting head of the first embodiment of figure 4.

Figure 8 shows a cross sectional perspective view of the cutting head, the impeller and drive shafts of the first embodiment of figure 4.

15 Figure 9 shows a perspective view of an alternative cutting head and impeller which can be used on the cutting apparatus of figures 4-5.

Figure 10 shows a perspective view of a second preferred embodiment of a cutting apparatus according to the invention.

20 Figure 11 shows a cross sectional view of the second embodiment of figure 10.

Figure 12 shows a detail of figure 11.

Figure 13 shows a cross sectional perspective view of the second embodiment of figure 10, with the cutting head lowered for removal from the impeller.

25 Figure 14 shows a perspective view of the second embodiment of figure 10, with the cutting head lowered and rotated away from the impeller.

Figure 15 shows a perspective view of a third preferred embodiment of a cutting apparatus according to the invention.

30 Figure 16 shows a perspective view of a fourth preferred embodiment of a cutting apparatus according to the invention.

Figure 17 shows a perspective view of a fifth preferred embodiment of a cutting apparatus according to the invention.

Figures 18-20 show top views of part of the cutting head and the impeller of an apparatus according to the invention to explain its operation.

5 Figure 21 shows a perspective view of a sixth preferred embodiment of a cutting apparatus according to the invention.

Figure 22 shows a cross sectional view of the cutting head and impeller of the sixth embodiment of figure 21.

10 Figure 23 shows a further alternative embodiment of a cutting head which can be used on apparatuses according to the invention.

Modes for carrying out the invention

15 The present invention will be described with respect to particular embodiments and with reference to certain drawings but the invention is not limited thereto but only by the claims. The drawings described are only schematic and are non-limiting. In the drawings, the size of some of the elements may be exaggerated and not drawn on scale for illustrative purposes. The dimensions and the relative dimensions do not necessarily correspond to actual reductions to practice of the invention.

20 Furthermore, the terms first, second, third and the like in the description and in the claims, are used for distinguishing between similar elements and not necessarily for describing a sequential or chronological order. The terms are interchangeable under appropriate circumstances and the embodiments of the invention can operate in other sequences than described or illustrated herein.

25 Moreover, the terms top, bottom, over, under and the like in the description and the claims are used for descriptive purposes and not necessarily for describing relative positions. The terms so used are interchangeable under appropriate circumstances and the embodiments of the invention described herein can operate in other orientations than described or illustrated herein.

30

Furthermore, the various embodiments, although referred to as "preferred" are to be construed as exemplary manners in which the invention may be implemented rather than as limiting the scope of the invention.

5 The term "comprising", used in the claims, should not be interpreted as being restricted to the elements or steps listed thereafter; it does not exclude other elements or steps. It needs to be interpreted as specifying the presence of the stated features, integers, steps or components as referred to, but does not preclude the presence or addition of one or more
10 other features, integers, steps or components, or groups thereof. Thus, the scope of the expression "a device comprising A and B" should not be limited to devices consisting only of components A and B, rather with respect to the present invention, the only enumerated components of the device are A and B, and further the claim should be interpreted as
15 including equivalents of those components.

Figures 1-3 respectively show a prior art impeller 30 and cutting head 20. The impeller 30 has a bottom plate 35 which is releasably fixed to a drive shaft of a prior art cutting apparatus for rotation inside the cutting head 20. The cutting head 20 is a cylindrical assembly comprising a top
20 ring 26, a bottom ring 29 and a plurality of cutting stations 27 held between these rings, each comprising one cutting element 28. The assembly is held together by a number of bolts and fixed to the frame base 10 of the machine. The cutting stations 27 are tiltable for adjusting the gap between the cutting element 28 and an opposite part at the rear of the subsequent
25 cutting station, i.e. for adjusting the thickness of the part which is cut off. The top sides of the cutting head 20 and impeller 30 are open. In use, product to be cut is supplied into the cutting head from this open top side, lands on the bottom plate 35 of the impeller and is moved towards the cutting elements 28 firstly by centrifugal force, which is imparted to the
30 product by the rotation of the impeller 30, and secondly by the paddles 34

of the impeller. In the prior art cutting apparatus, the cutting head 20 is stationary.

The cutting apparatus shown in figures 4-8 is a first embodiment of a cutting apparatus according to the invention. It comprises a base 100 which carries a rotatable cutting head 200 and an impeller 300, adapted for rotating concentrically within the cutting head. A first drive mechanism, which is constituted by a first drive shaft 301, drive belt 302 and motor 303, is provided for driving the rotation of the impeller 300. A second drive mechanism, which is constituted by a second drive shaft 201, drive belt 202 and motor 203, is provided for driving the rotation of the cutting head. The first and second drive shafts are concentric. The second drive shaft 201 which drives the cutting head 200 is rotatably mounted by means of bearings 104, 105 inside a stationary outer bearing housing 103, which forms part of the base 100. The first drive shaft 301 which drives the impeller is rotatably mounted by means of bearings 106, 107 inside the first drive shaft 201. As shown, these bearings 104-107 are tapered roller bearings, slanting in opposite directions, which is preferred in view of withstanding the forces which occur during operation of the apparatus. Alternatively, angular contact bearings could be used, or any other bearings deemed suitable by the person skilled in the art.

The base 100 comprises an arm 101, which is rotatably mounted on a post 102, so that the cutting head 200 and impeller 300 can be rotated away from the cutting position for cleaning, maintenance, replacement etc.

Figures 6-8 respectively show the impeller 300 and cutting head 200 fitted on the apparatus of figures 4-5. The impeller 300 is releasably fixed to the first drive shaft 301 for rotation inside the cutting head 200. The cutting head 200 is a cylindrical assembly comprising a top ring 206, a bottom plate 205 and a plurality of cutting stations 207 held between these two parts, each comprising one cutting element 208. The assembly is held together by a number of bolts and releasably fixed to the second drive shaft 201. The cutting stations 207 are tiltable for adjusting the gap

between the cutting element 208 and an opposite part at the rear of the subsequent cutting station, i.e. for adjusting the thickness of the part which is cut off. The top sides of the cutting head 200 and impeller 300 are open. In use, product to be cut is supplied into the cutting head from this open
5 top side, lands on the bottom plate 305 of the impeller and is moved towards the cutting elements 208 firstly by centrifugal force, which is imparted to the product by the rotation of the impeller 300, and secondly by the paddles 304 of the impeller.

The cutting head 200 is fitted with cutting elements 208, for
10 example blades which make straight cuts in the product, for example to make potato chips. As an alternative, corrugated cutting elements could be fitted in order to make for example crinkle cut potato chips or shreds.

Figure 9 shows an alternative embodiment of a cutting head 400 with an adapted impeller 410 which is also capable of being used on the
15 apparatus of figures 4-5. The cutting head and impeller again are both rotatable and are driven by means of concentric shafts in the same way as described above. The cutting stations 401 in this embodiment comprise each a larger blade 402 and a number of smaller, so-called julienne tabs 403 extending at an angle thereto, in particular substantially perpendicular
20 thereto. In the embodiment shown, the julienne tabs 403 are welded onto the larger blades 402, but they could also be removably fixed thereto. In particular, in the embodiment shown the julienne tabs 403 are fixed to and extend perpendicular to the bevel of the larger blades 402, but they could also be fixed to the larger blades 402 behind the bevel. The front cutting
25 edges of the julienne tabs 403 are slightly behind the front cutting edge of the larger blade 402, all at the same distance. Alternatively, they could also be located at varying distances from the front cutting edge of the larger blade 402, for example in a staggered or alternating configuration. The julienne tabs 403 are stabilised by means of slots 404 in the
30 subsequent cutting station, so that during operation stresses can be relieved and the desired cut can be better maintained. The slots 404

extend a given distance into the rear end of the cutting stations 401 to accommodate for the variable positions of the julienne tabs 403 upon pivoting the cutting stations 401 for varying the gap. With this cutting head 400, the product is cut in two directions at once. It can for example be
5 used to cut French fries from potatoes or to cut lettuce.

In further alternatives, cutting stations can be used with cutting edges for milling or comminuting products (e.g. salt, spices) or viscous liquids (e.g. butters, spreads). With these cutting stations, the apparatus can also be used for manufacturing pharmaceutical products like for
10 example ointments.

In further alternatives, cutting stations can be used with grating surfaces for making grated cheese, or with any other cutting elements known to the person skilled in the art. The cutting apparatus of figures 4-5 can even be used with the prior art cutting head and impeller of figures 1-
15 3.

Figures 21 and 22 show an alternative embodiment of an impeller 420 which can be used on the apparatus of figures 4-5 with the same cutting head 200. The impeller 420 comprises a feed tube 421 which starts vertically in the centre of the impeller and bends towards the cutting head
20 200. This impeller 420 is intended for products for which it is desired to feed them towards the cutting head 200 in a directed way, such as, for example, products with an elongated shape of which it is desired their shorter sides face the cutting elements 208 and they are cut into chips having a more circular shape. The mouth of the feed tube can also be
25 oriented at an angle with respect to the cutting elements 208, so that the products are cut into chips having a more oval shape. The impeller 420 is for example highly suitable for cutting larger, elongated potatoes into circular chips or for cutting onions into onion rings.

The cutting apparatus shown in figures 10-14 has many features in
30 common with the cutting apparatus shown in figures 4-5. As a result, only the differences will be explained in detail.

The cutting apparatus shown in figures 10-14 is mainly different in the driving mechanisms used to drive the impeller 500 and the cutting head 600. For both, an in line drive mechanism is used, i.e. the impeller 500 is directly fixed to the shaft of the motor 503 and the cutting head 600 is directly fixed to the shaft of the motor 603. This has the advantage that any intermediate drive components, such as the driving belts 202, 302 and the concentric shafts 201, 202 of the apparatus of figures 4-5 are avoided, which simplifies the construction. The concentric rotation of the impeller 500 inside the cutting head 600 is stabilised by means of a spring-loaded pin 501 which fits into a tapered hole 601 in the centre of the cutting head 600.

The cutting head 600 is in this embodiment an assembly of a top ring 606, cutting stations 607 and a spider support 609 at the bottom. The cutting stations 607 are held between the top ring 606 and the spider support 609 like in the above described embodiment. The spider support 609 is used instead of a full bottom plate in order to save weight. The spider support can be connected to the shaft of the motor 603 by means of notches which are engaged by pins on the shaft. This can be a quick release engagement which can be fixed/loosened by for example turning the spider support 609 over $+5^{\circ}/-5^{\circ}$ with respect to the motor shaft. Of course, the spider support 609 could also be bolted to the motor shaft, or releasably fixed by any other means known to the person skilled in the art.

In this embodiment, the base 110 comprises a vertical post 111 with a fixed top arm 112 on which the impeller motor 503 is mounted with the shaft pointing downwards. The cutting head motor 603 is mounted on the post 111 with the shaft pointing upwards by means of a vertically movable and horizontally rotatable arm 113. In this way, the cutting head 600 can be removed from the impeller 500 for maintenance, replacement, etc. by subsequently moving the arm 113 downwards (fig. 13) and rotating it in a horizontal plane (fig. 14).

The cutting apparatus shown in figure 15 is the same as the one of figures 4-5, but the cutting head 200 and the impeller 300 are oriented for rotation around a horizontal axis and are mounted adjacent a dicing unit 430. For dicing product by means of this apparatus, the cutting head 200
5 can here be locked to the base 100 by means of a releasable locking mechanism (not shown) to make it stationary. For dicing, the cutting stations 207 can all be tilted to a non-cutting position (zero gap) except for the one located at the dicing unit 430. A dicing unit is otherwise known in the art and therefore needs no further description here. So in this
10 embodiment, the apparatus is convertible between a first mode of operation, namely with a stationary cutting head adjacent a dicing unit, and a second mode of operation with a rotating cutting head.

The cutting apparatus shown in figure 16 is similar to that of figures 4-5 in that it has the same cutting head 200 and impeller 300 with
15 concentrical drive shafts, mounted on a base 100 comprising an arm 101 which is rotatably mounted on a post 102. The drive mechanisms for the cutting head and the impeller are however different in the aspect that they comprise a shared motor 120 with two shafts: a first shaft 121 running the drive belt 302 for the impeller 300 and a second shaft 122 running the
20 drive belt 202 for the cutting head 200. These shafts 121, 122 are internally coupled to each other by means of a gear mechanism which sets a predetermined ratio of the rotational speeds of the shafts and the rotational relationship, i.e. whether the cutting head and the impeller rotate in the same direction or not. So in this embodiment there is a fixed ratio
25 between the first rotational speed of the impeller 300 and the second rotational speed of the cutting head 200, which means that this apparatus is configured for always cutting the same product or at least products for which the fixed ratio is optimal.

The cutting apparatus shown in figure 17 is similar to that of figures
30 4-5 in that it has the same cutting head 200 and impeller 300 with concentrical drive shafts, mounted on a top part 131 of a base 130 which

is tiltably fixed on a vertical post 132. In this way, the top part 131 carrying the cutting head 200 and impeller 300 can be tilted as a whole, so that the angle at which the cutting head 200 and the impeller 300 rotate is adaptable to the situation.

5 Below, the operation of the cutting apparatus of the invention will be discussed in general by reference to figures 18-20. For the sake of simplicity, the reference numbers of the first embodiment of figures 4-8 are used, but note that each of these situations can be applied to each of the above described embodiments as well as any other variations utilizing the principles of the present invention. In these figures, the cutting elements
10 208 of the cutting head 200 are oriented to impart cutting action in counterclockwise direction, i.e. the cutting elements cut through the product in counterclockwise direction or, alternatively stated, the product passes the cutting elements in clockwise direction. This is the mode of operation which is used in the art (with stationary cutting heads), but it is
15 evident that the orientation of the cutting elements can be turned around to impart cutting action in clockwise direction. The arrows v_{CH} and v_{IMP} on these figures respectively represent the rotational speed of the cutting head and the rotational speed of the impeller.

20 In the situation of figure 18, the impeller 300 and the cutting head 200 rotate in the same direction, namely both clockwise. They rotate at different rotational speeds, i.e. the cutting head is not stationary with respect to the impeller. The first rotational speed v_{IMP} of the impeller 300 is greater than the second rotational speed v_{CH} of the cutting head 200, so
25 that the paddles 304 of the impeller move the product towards the cutting elements 208. The first rotational speed of the impeller 300 sets the centrifugal force exerted on the product, i.e. the force with which the product is pressed against the interior of the cutting stations 207. The difference in rotational speed sets the cutting velocity with which the
30 cutting elements 208 cut through the product, which is pushed towards them by means of the paddles of the impeller 304.

In the situation of figure 19, the impeller 300 and the cutting head 200 rotate in opposite directions, namely the impeller 300 rotates clockwise and the cutting head 200 rotates counterclockwise. In this situation, the first and second rotational speeds v_{IMP} and v_{CH} can be equal or different in absolute value. The first rotational speed v_{IMP} of the impeller 300 sets the centrifugal force. The cutting velocity is related to the sum of the absolute values of the rotational speeds v_{CH} and v_{IMP} , as their direction is opposite.

In the situation of figure 20, the impeller 300 and the cutting head 200 rotate in the same direction, namely both counterclockwise, with the impeller 300 at a smaller rotational speed than the cutting head 200. The first rotational speed v_{IMP} of the impeller 300 sets the centrifugal force. As the first rotational speed v_{IMP} is smaller than the second rotational speed v_{CH} , the cutting elements 208 move towards the paddles 304, so towards the product to be cut. The cutting velocity is determined by the difference between the first and second rotational speeds.

By way of example, some preferred settings for cutting potatoes are given. Table 1 below shows the relationship between the impeller rotational speed for a 178 mm radius and the centrifugal force experienced by potatoes of different weights. At 260 RPM, the centrifugal acceleration (g-force) is 131.95 m/s^2 ($\cong 13 \text{ g}$) which corresponds to the centrifugal forces in the second column for the weights given in the first column; at 230 RPM, the centrifugal acceleration (g-force) is 103.26 m/s^2 ($\cong 10 \text{ g}$) which corresponds to the centrifugal forces in the third column for the weights given in the first column.

POTATO WEIGHT	IMPELLER RPM	
	CENTRIFUGAL ACCELERATION	CENTRIFUGAL ACCELERATION
	131.95 m/s^2 ($\cong 13 \text{ g}$) @ 260 RPM & 178 mm	103.26 m/s^2 ($\cong 10 \text{ g}$) @ 230 RPM & 178 mm

	RADIUS	RADIUS
0.70 kg	92 N	72 N
0.45 kg	59 N	46 N
0.30 kg	40 N	31 N
0.20 kg	26 N	21 N
0.10 kg	13 N	10 N

Table 1

It has been found that the impeller rotational speed is preferably controlled such that the g-force experienced by product being cut is in the range of 1 to 50 g's (1 g = 9.8 m/s²), although even higher g-forces may be used, for example in comminuting.

For cutting potatoes, a range of 3 to 30 g's appears to yield the best results.

For cutting potatoes, the cutting velocity is preferably in the range of .3 to 4.8 m/s, more preferably in the lower half of this range.

For cutting or shredding cheese products, also a range of 3 to 30 g's appears to yield the best results.

For cutting or shredding cheese products, the cutting velocity is preferably in the range of .3 to 5.5 m/s.

Importantly, with the apparatus and method of the invention, the centrifugal force can be reduced with respect to the prior art with a stationary cutting head. In such prior art apparatuses, when cutting cheese products the impeller is rotated at a relatively high speed (e.g. 400 RPM) in order to obtain the desired cutting velocity, but at such speeds the cheese products may be undesirably compressed against the interior of the cutting head. So in order to obtain a good quality of cutting, the cheese product needed to be cooled to a temperature of -4°C to harden the product and avoid compression. With the apparatus of the invention, the centrifugal force can be reduced and the cutting velocity set independently therefrom, so that the cutting operation can occur at higher temperatures,

i.e. temperatures of -3°C or above, e.g. at 10°C , reducing the extent of cooling needed prior to cutting.

Examples of other products which can be cut in a more advantageous way with the apparatus and method of the invention are nut products, e.g. almonds, peanuts (e.g. to manufacture peanut butter) or other nuts; root products, e.g. ginger, garlic, or other; and also other products such as e.g. orange peel.

Figure 23 shows a further alternative embodiment of a cutting head 250 which can be used on apparatuses according to the invention, for example together with the same impeller 300 described above. The cutting head 250 comprises cutting stations 257 which have cutting elements 258, 259 at both ends. These cutting stations 257 are tiltable for setting the gap and also for setting the direction in which the cutting head cuts, i.e. in clockwise or counterclockwise directions. In other words, this cutting head 257 is capable of cutting products by rotation in either direction, provided that the cutting stations are correctly set.

In further embodiments (not shown), the impeller drive shaft could also be made hollow, for example for accommodating a large bolt with which the impeller is fixed to the impeller drive shaft, or for connecting a liquid supply and supplying a liquid (e.g. water) to the cutting head from the bottom side through the impeller drive shaft, or both, in which case the bolt would also be hollow.

Claims

1. Apparatus for cutting products, comprising:
- a base;
 - a cutting head with at least one cutting element along the circumference of the cutting head for cutting products fed into the cutting head, the cutting head being rotatably fitted to the base;
 - an impeller adapted for rotating concentrically within the cutting head to urge products fed into the cutting head towards the circumference of the cutting head by means of centrifugal force;
 - a first drive mechanism for driving the rotation of the impeller at a first rotational speed setting the centrifugal force; and
 - a second drive mechanism for driving the rotation of the cutting head at a second rotational speed, determined such with respect to the first rotational speed that the product is cut by the at least one cutting element at a predetermined cutting velocity.
2. Apparatus according to claim 1, wherein the first and second drive mechanisms are provided with controls for controlling the first and second rotational speeds within respectively a first range and a second range.
3. Apparatus according to claim 1 or 2, wherein the first drive mechanism comprises a first drive shaft by which the impeller is driven and the second drive mechanism comprises a second drive shaft by which the cutting head is driven, the second drive shaft being hollow and the first drive shaft being rotatably mounted within the second drive shaft.
4. Apparatus according to any one of the preceding claims, wherein the first and second drive mechanisms have separate motors.

5. Apparatus according to claim 4, wherein the impeller is directly driven by a first motor of the first drive mechanism and the cutting head is directly driven by a second motor of the second drive mechanism.

5 6. Apparatus according to claim 5, wherein the base comprises a post with a first arm carrying the first motor with the impeller and a second arm carrying the second motor with the cutting head, the second arm being movably mounted to the post in such a way that the cutting head can be removed from around the impeller.

10 7. Apparatus according to claim 5 or 6, wherein the rotation of the impeller inside the cutting head is stabilised by means of a spring-loaded pin on the impeller which fits into a tapered hole in the centre of the cutting head.

15 8. Apparatus according to any one of the preceding claims, wherein the first and second drive mechanisms together comprise a shared motor and a gearbox, adapted for setting the difference between the first rotational speed and the second rotational speed.

20 9. Apparatus according to any one of the preceding claims, wherein the cutting head and the impeller are oriented to rotate around a vertical axis.

25 10. Apparatus according to any one of the claims 1-8, wherein the cutting head and the impeller are oriented to rotate around a horizontal axis.

30 11. Apparatus according to any one of the preceding claims, wherein the cutting head and the impeller are mounted on a tiltable part of

the base, by means of which the rotation axis of the cutting head and the impeller can be tilted to different angles.

12. Apparatus according to any one of the preceding claims, further
5 comprising a releasable locking mechanism for releasably fixing the cutting head to the base.

13. Apparatus according to any one of the preceding claims,
wherein each cutting element comprises a larger blade and a number of
10 smaller blades extending at an angle to the larger blade.

14. Apparatus according to any one of the preceding claims,
wherein each cutting element comprises a larger blade and a number of
julienne tabs extending substantially perpendicular to the larger blade and
15 wherein each cutting station comprises a rear end with slots for holding and stabilising the julienne tabs of the cutting element of the adjacent cutting station.

15. Apparatus according to any one of the preceding claims,
20 wherein the impeller comprises a feed tube which starts vertically in the centre of the impeller and bends towards the cutting head.

16. Apparatus according to any one of the preceding claims,
wherein the cutting head and the impeller are configured for rotating in the
25 same direction.

17. Apparatus according to any one of the preceding claims,
wherein the cutting head and the impeller are configured for rotating in
opposite directions.

18. Apparatus according to any one of the preceding claims, configured for cutting potatoes, wherein controls are provided for setting the predetermined difference between the impeller and cutting head rotational speeds such that a cutting velocity below 4.8 m/s is obtained.

5

19. Apparatus according to any one of the preceding claims, configured for cutting potatoes, wherein controls are provided for setting the impeller rotational speed such that the potatoes are cut while experiencing a g-force of 3 to 30 g's.

10

20. Apparatus according to any one of the preceding claims, configured for cutting cheese products, wherein controls are provided for setting the predetermined difference between the impeller and cutting head rotational speeds such that a cutting velocity below 5.5 m/s is obtained.

15

21. Apparatus according to any one of the preceding claims, configured for cutting cheese products, wherein controls are provided for setting the impeller rotational speed such that the cheese products are cut while experiencing a g-force of 3 to 30 g's.

20

22. Method for cutting a product, comprising the steps of:

- feeding the product to a cutting head which has at least one cutting element along the circumference of the cutting head for the cutting product, which is rotatably fitted to a base and which comprises an impeller adapted for rotating concentrically within the cutting head to urge the product towards the circumference of the cutting head by means of centrifugal force;
- rotating the impeller at a first rotational speed setting the centrifugal force;

25

30

- rotating the cutting head at a second rotational speed, determined such with respect to the first rotational speed that the product is cut by the at least one cutting element at a predetermined cutting velocity.

5

23. Method according to claim 22, further comprising the step of controlling the first and second rotational speeds.

24. Method according to claim 22 or 23, wherein the impeller is driven by means of a first drive shaft and the cutting head is which the cutting head is driven, the second drive shaft being hollow and the first drive shaft being rotatably mounted within the second drive shaft.

25. Method according to any one of the claims 22-24, wherein the first and second rotational speeds are obtained by means of separate motors.

26. Method according to claim 25, wherein the impeller and the cutting head are directly driven by their respective motors.

20

27. Method according to claim 26, further comprising the step of removing the cutting head from around the impeller.

28. Method according to claim 26 or 27, wherein the rotation of the impeller inside the cutting head is stabilised by means of a spring-loaded pin on the impeller which fits into a tapered hole in the centre of the cutting head.

29. Method according to any one of the claims 22-28, wherein the first and second rotational speeds are obtained by means of a shared motor and a gearbox.

30

30. Method according to any one of the claims 22-29, wherein the cutting head and the impeller are oriented to rotate around a vertical axis.

5 31. Method according to any one of the claims 22-29, wherein the cutting head and the impeller are oriented to rotate around a horizontal axis.

10 32. Method according to any one of the claims 22-31, further comprising the step of tilting the cutting head and the impeller, such that their rotation axis is set to a predetermined angle.

15 33. Method according to any one of the claims 22-32, further comprising the step of releasably fixing the cutting head to the base by means of a releasable locking mechanism.

20 34. Method according to any one of the claims 22-33, further comprising the step of fitting the cutting head with cutting elements which each comprise a larger blade and a number of smaller blades extending at an angle to the larger blade.

25 35. Method according to any one of the claims 22-34, further comprising the steps of fitting the cutting head with cutting elements which each comprise a larger blade and a number of julienne tabs extending substantially perpendicular to the larger blade and stabilising the julienne tabs of each cutting element in slots of a rear end of the adjacent cutting station.

30 36. Method according to any one of the claims 22-35, further comprising the step of using an impeller which comprises a feed tube

which starts vertically in the centre of the impeller and bends towards the cutting head.

37. Method according to any one of the claims 22-36, wherein the
5 cutting head and the impeller are rotated in the same direction.

38. Method according to any one of the claims 22-37, wherein the cutting head and the impeller are rotated in opposite directions.

10 39. Method according to any one of the claims 22-38, wherein the product is potatoes.

40. Method according to claim 39, wherein the predetermined difference between the first rotational speed and the second rotational
15 speed is set for obtaining a cutting velocity below 4.8 m/s.

41. Method according to claim 38 or 39, wherein the first rotational speed is controlled such that the potatoes are cut while experiencing a g-force of 3 to 30 g's.
20

42. Method according to any one of the claims 22-38, wherein the product is cheese.

43. Method according to claim 42, wherein the predetermined difference between the first rotational speed and the second rotational
25 speed is set for obtaining a cutting velocity below 5.5 m/s.

44. Method according to claim 42 or 43, wherein the first rotational speed is controlled such that the cheese is cut while experiencing a g-force of 3 to 30 g's.
30

45. Method according to claim 42, 43 or 44, wherein the cheese is cut at a temperature above -3°C .

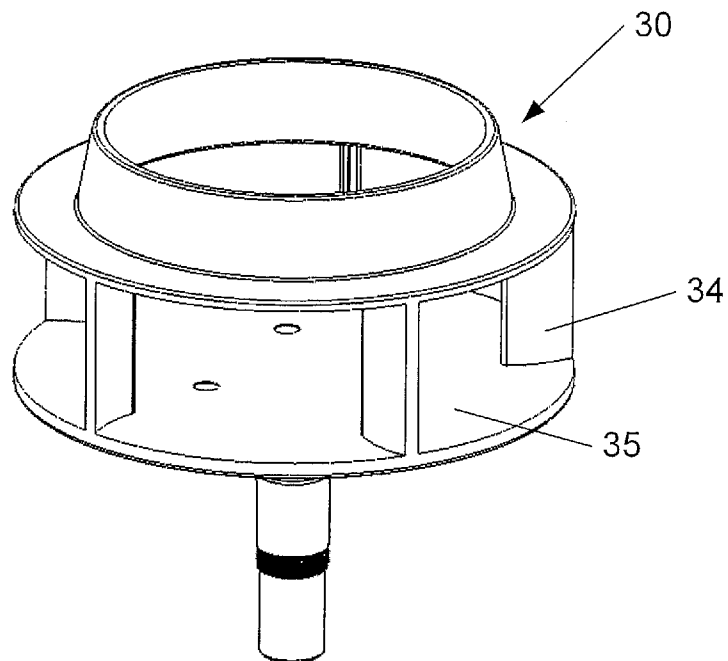


Fig. 1
(prior art)

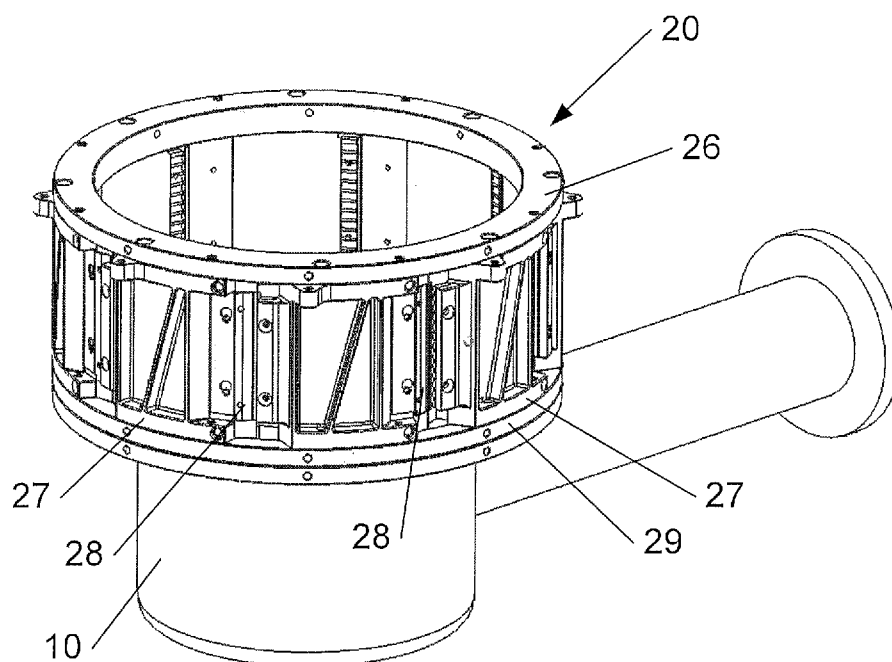


Fig. 2
(prior art)

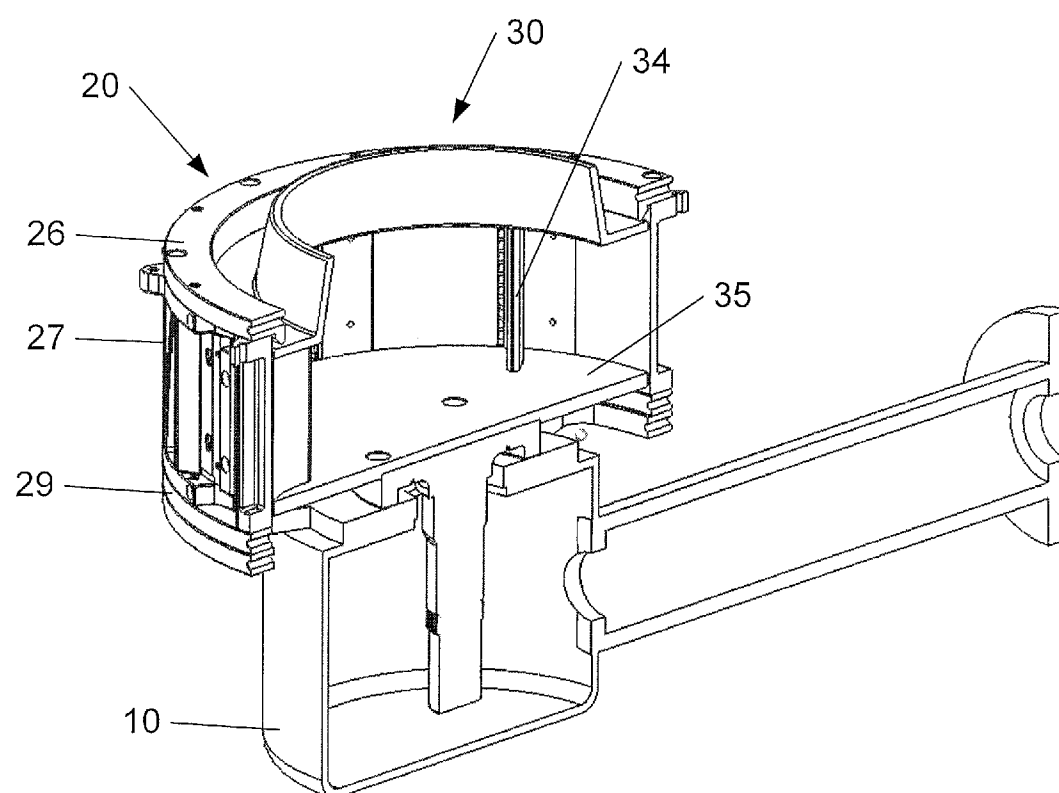
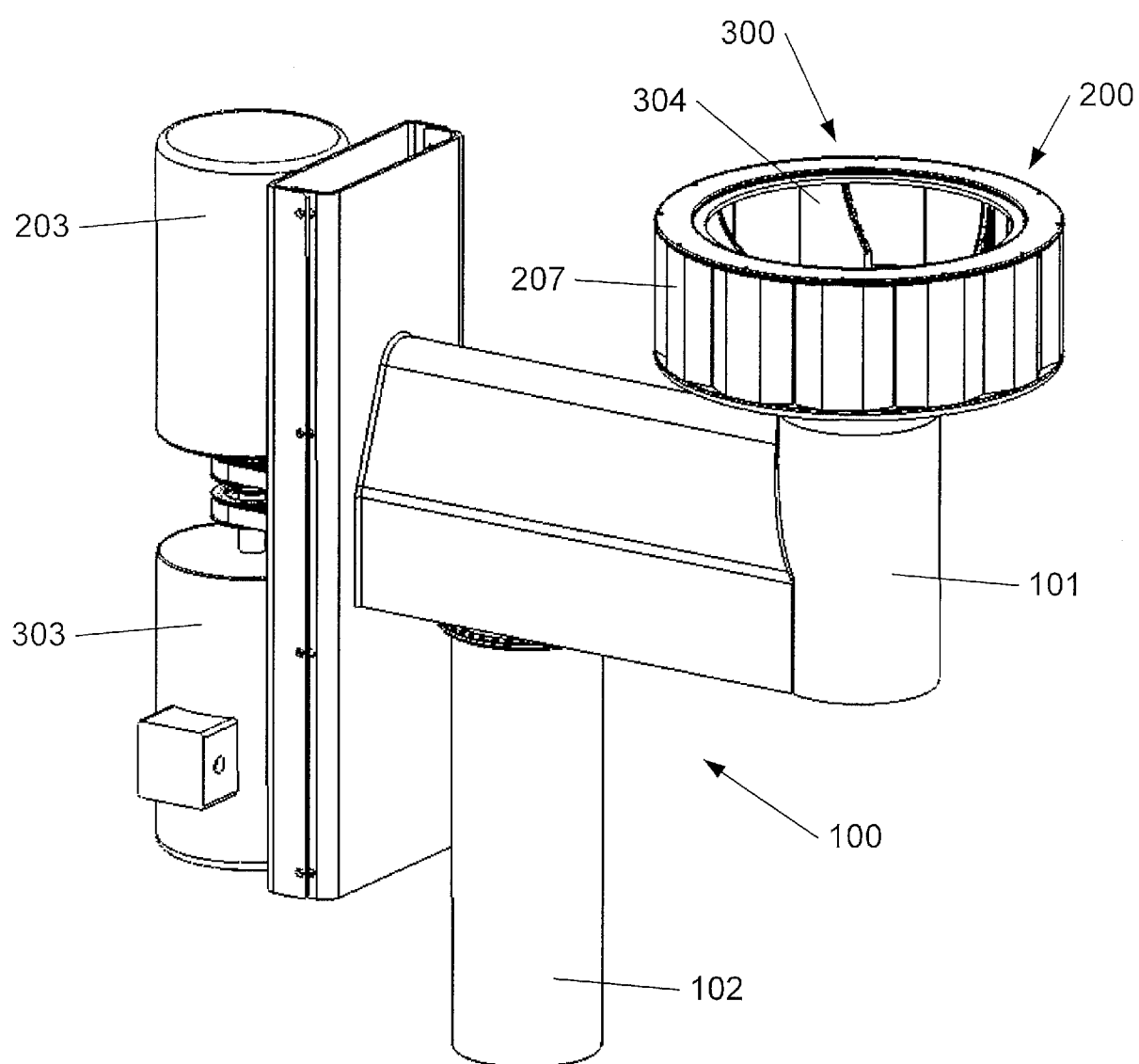
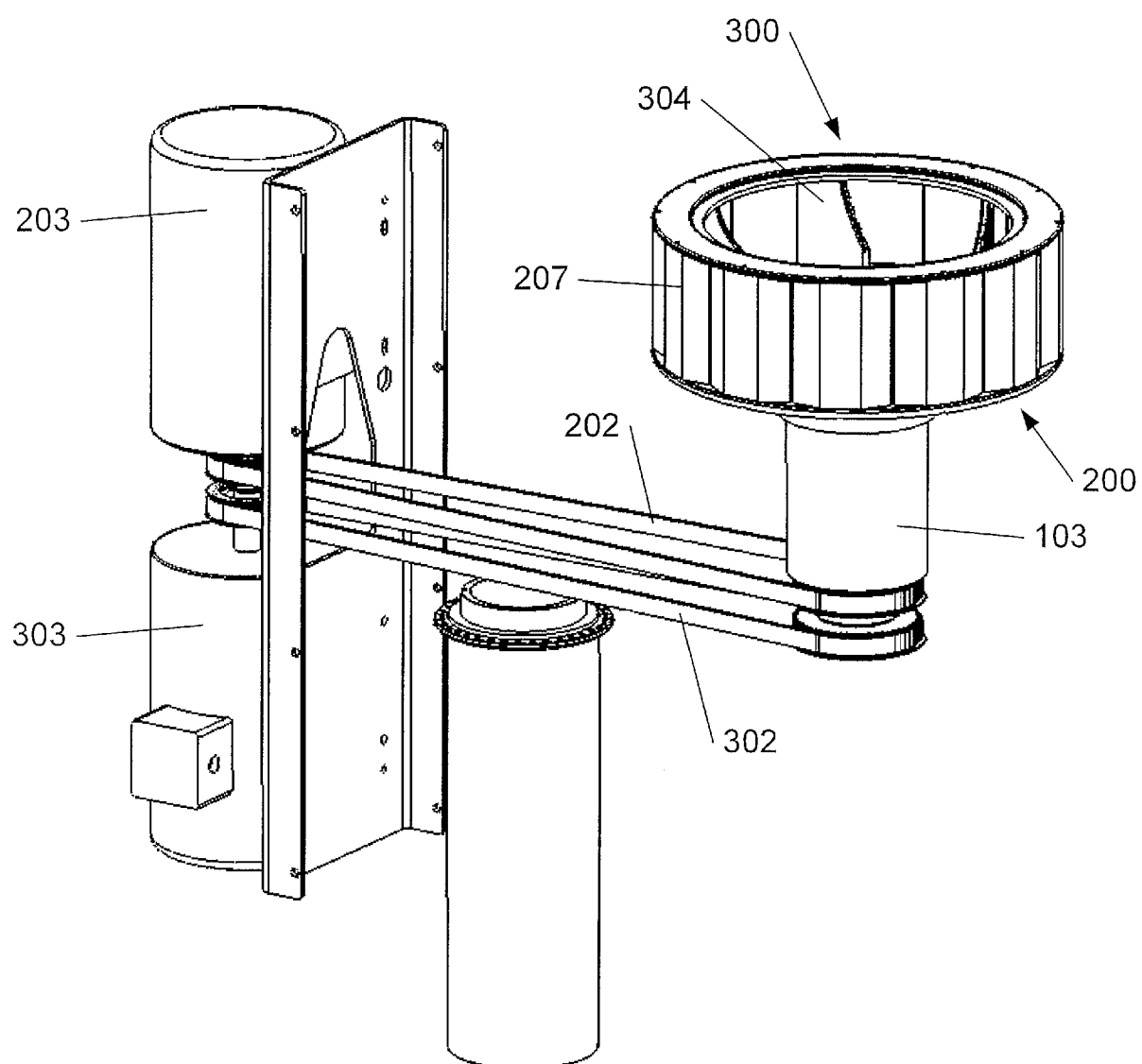
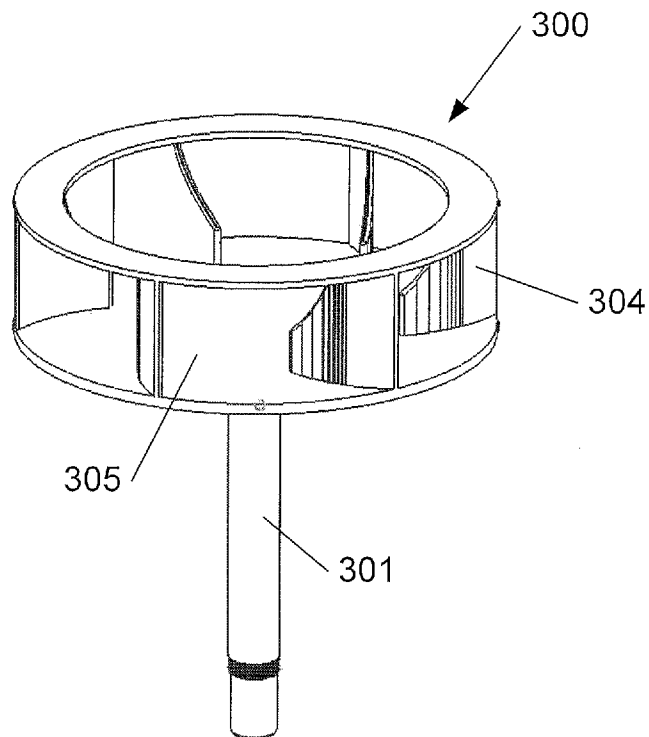
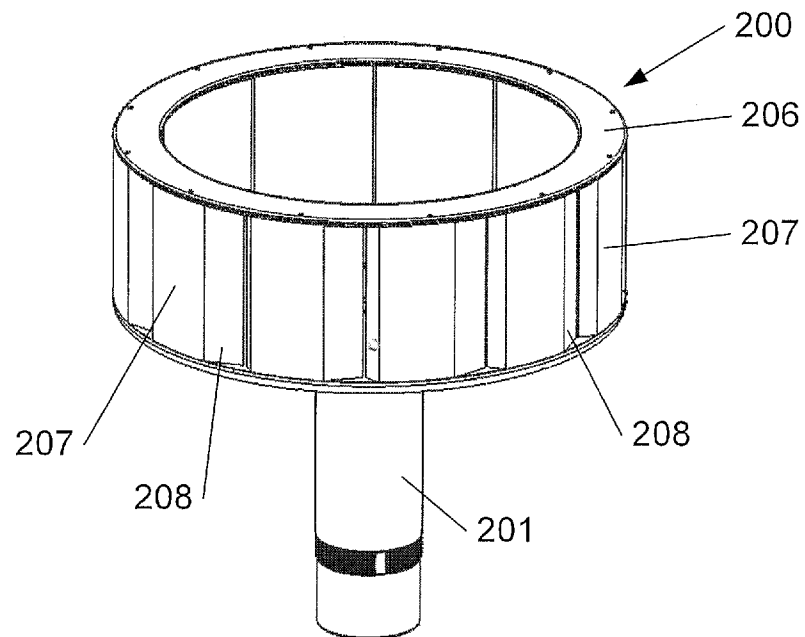


Fig. 3
(prior art)

***Fig. 4***

***Fig. 5***

***Fig. 6******Fig. 7***

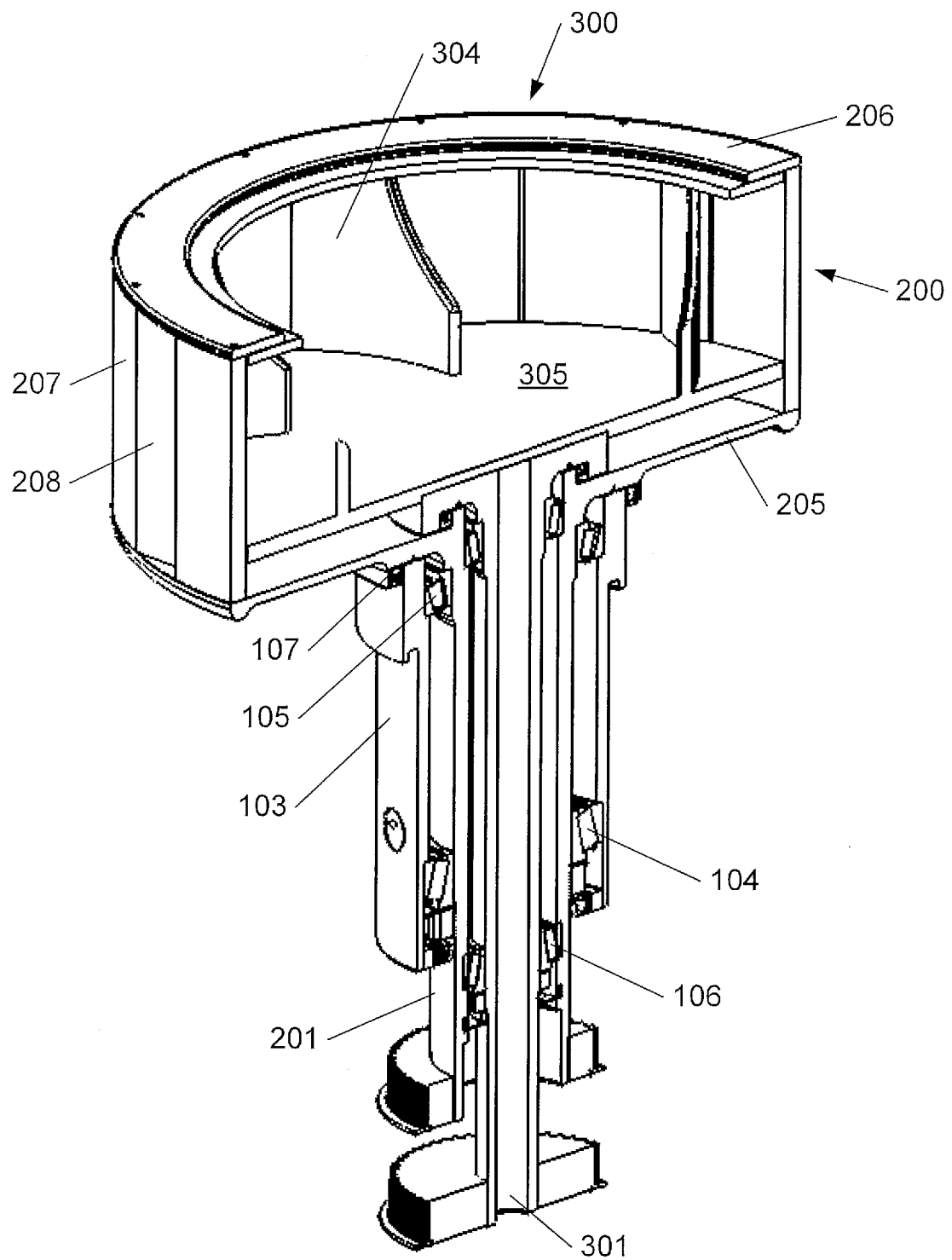
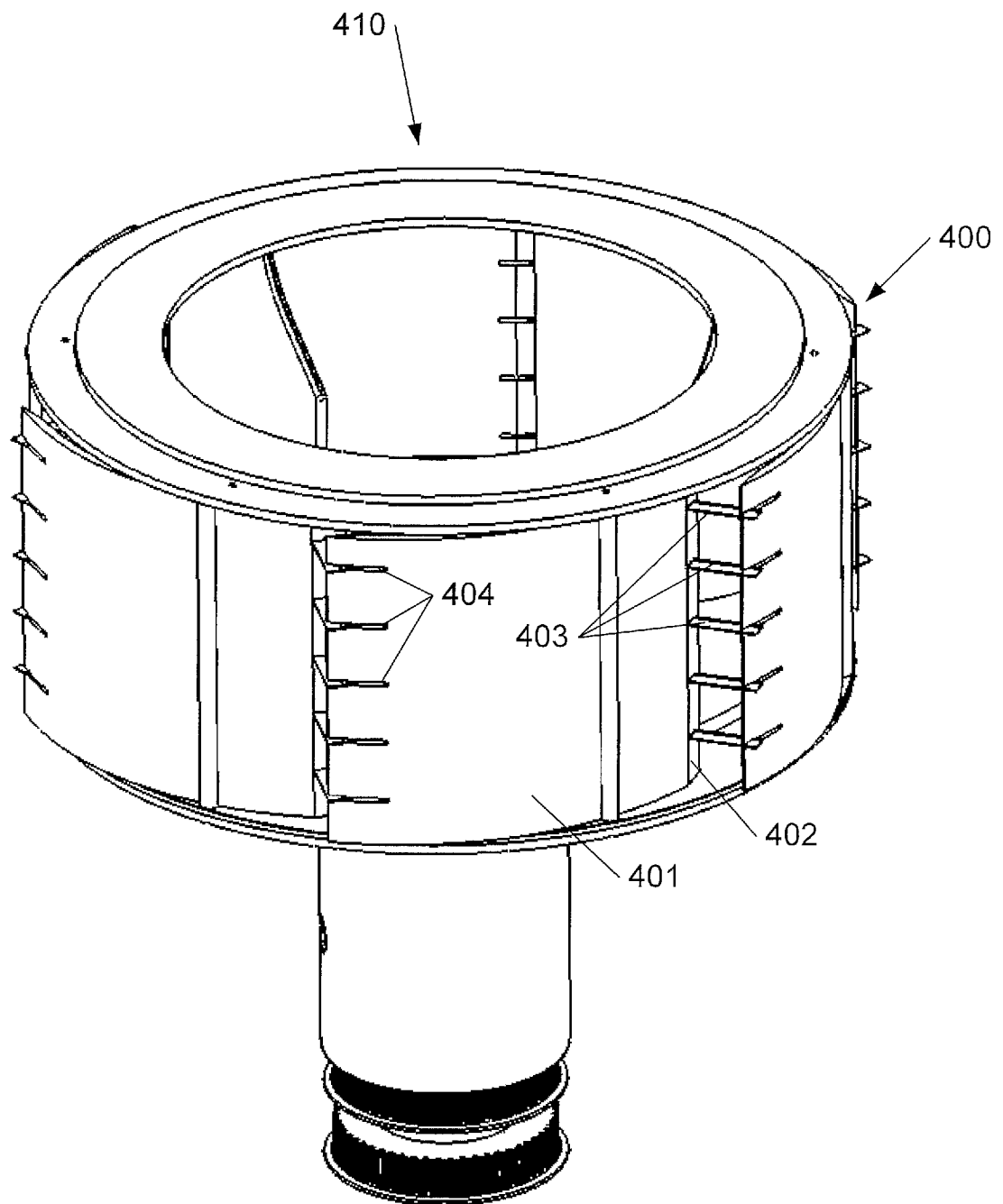
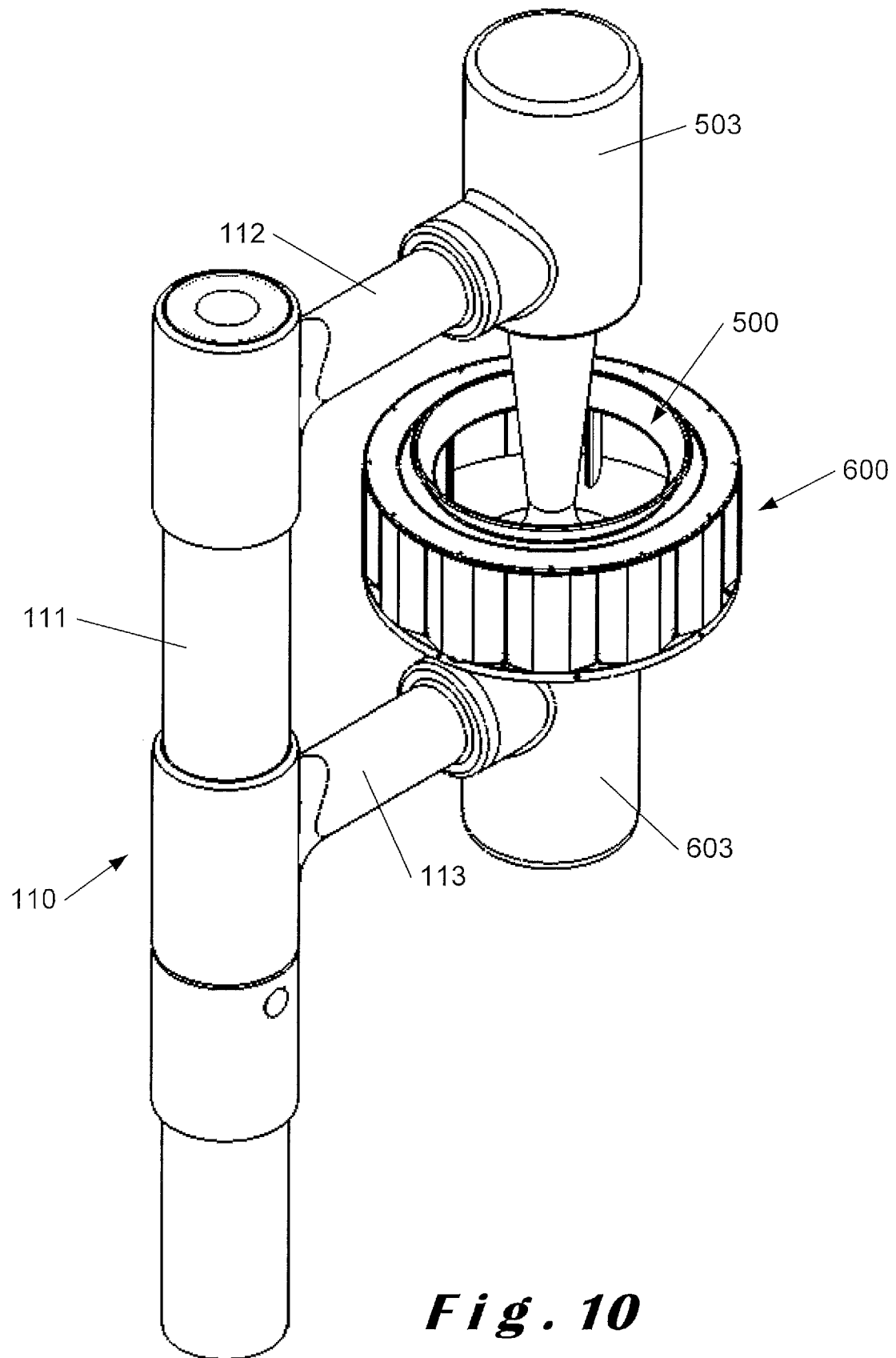
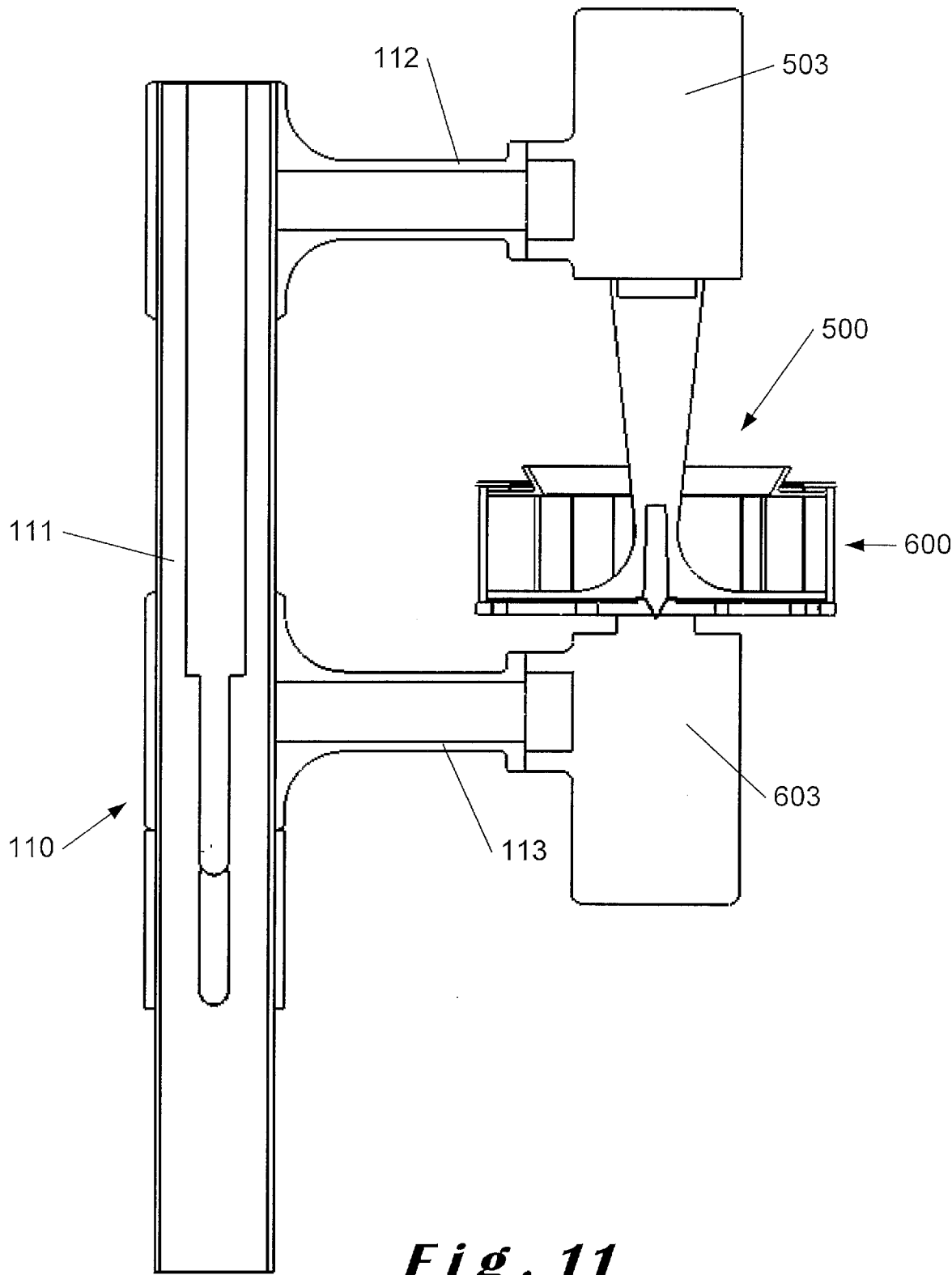
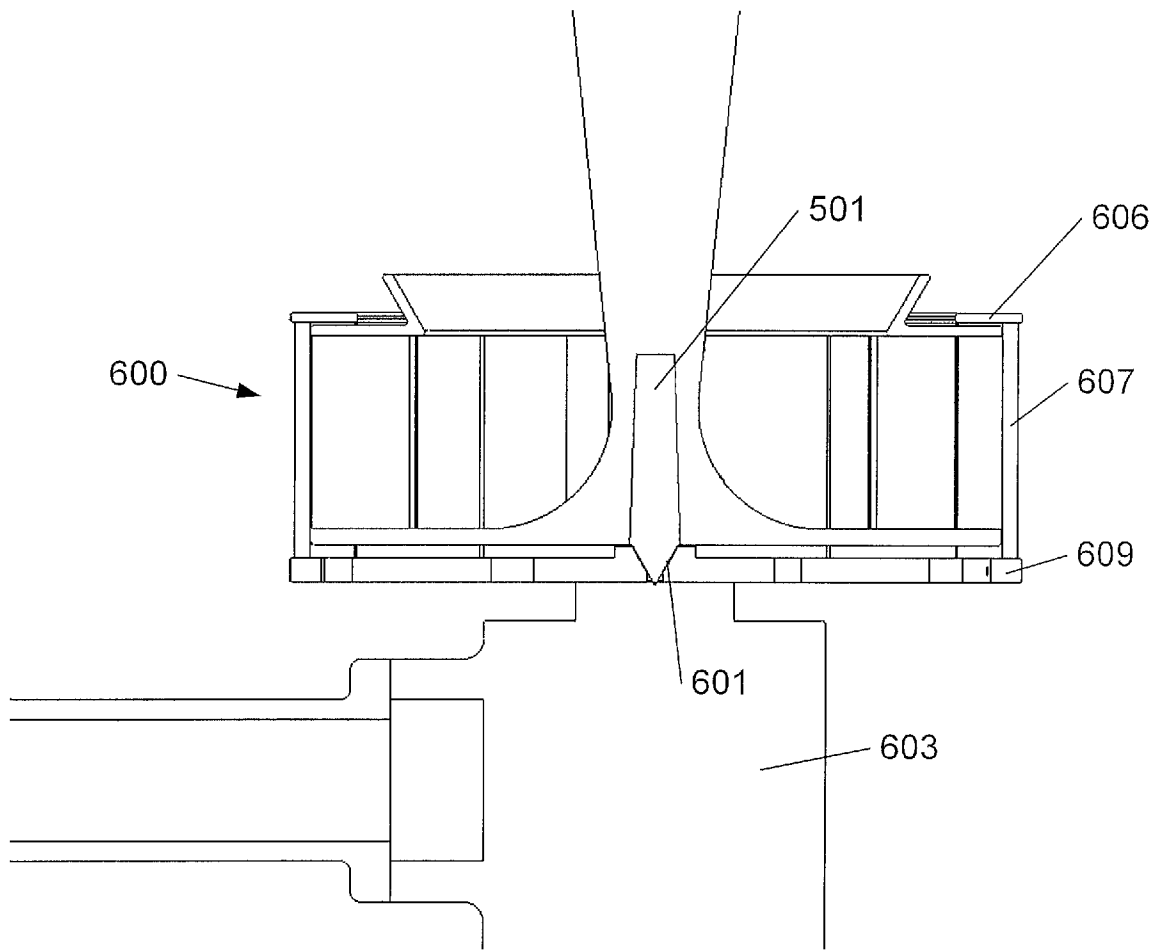


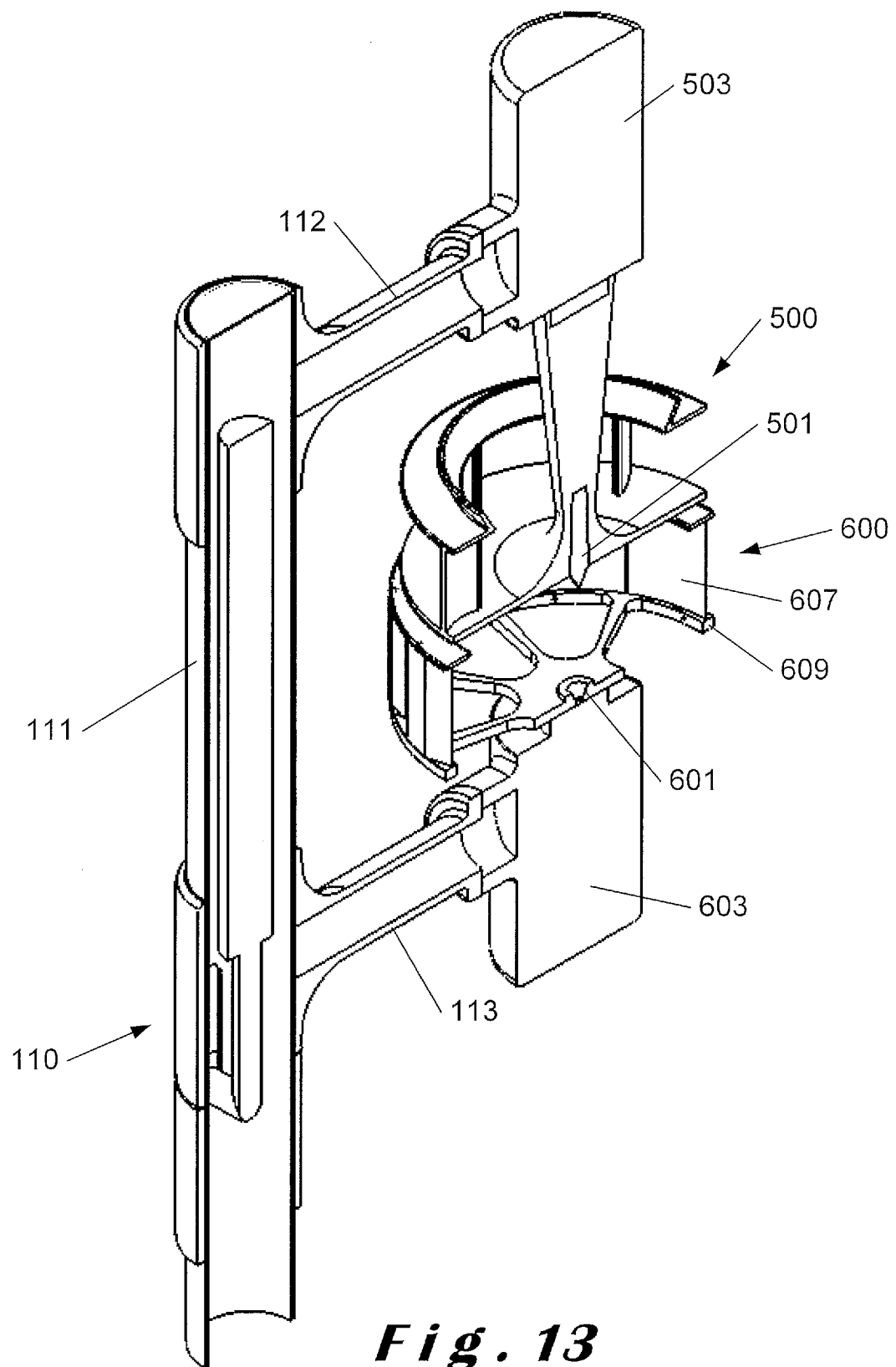
Fig. 8

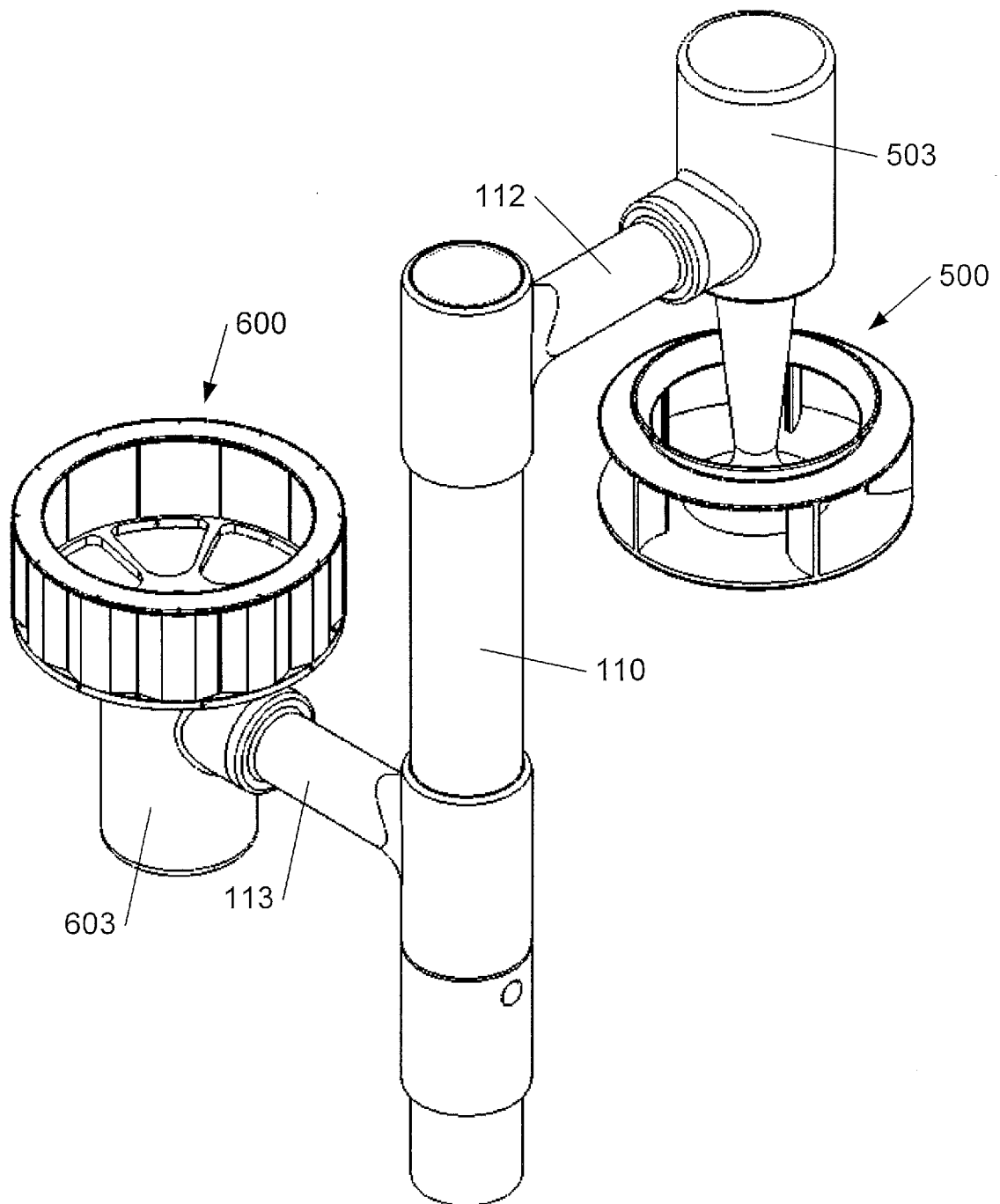
***Fig. 9***

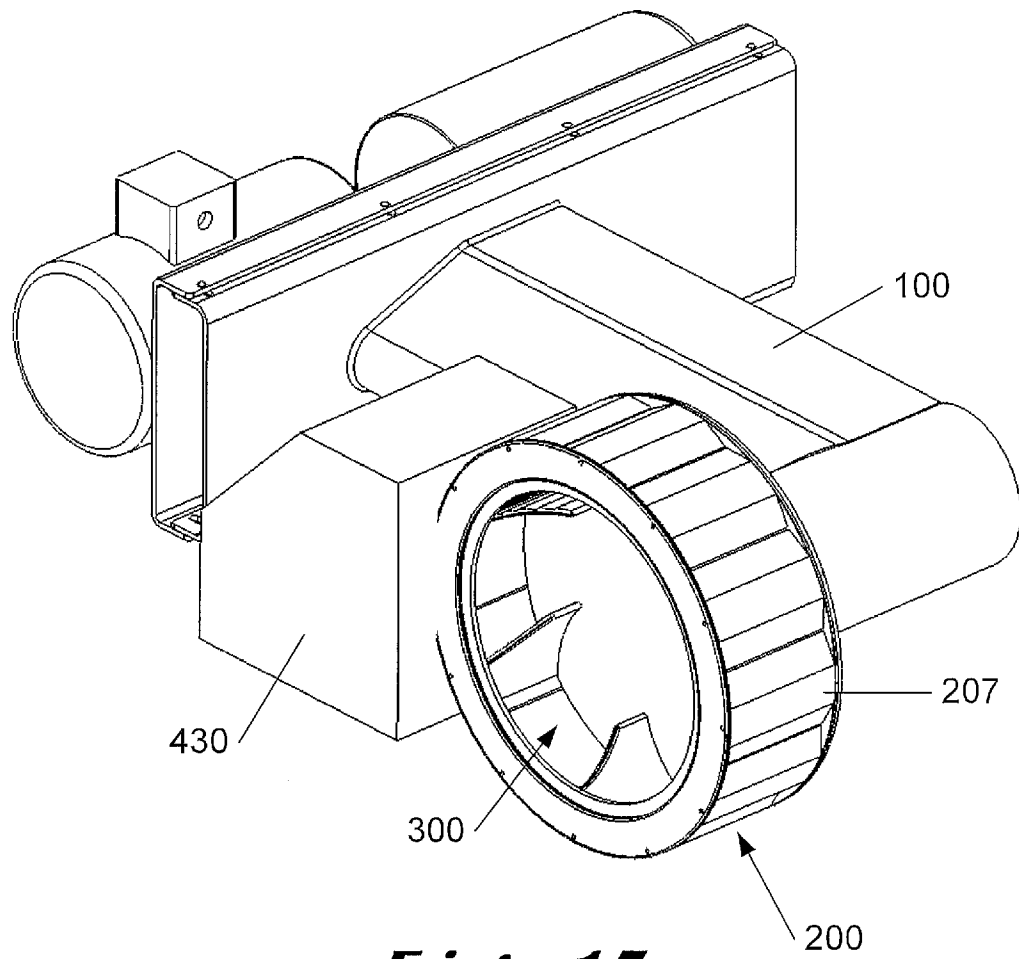
***Fig. 10***

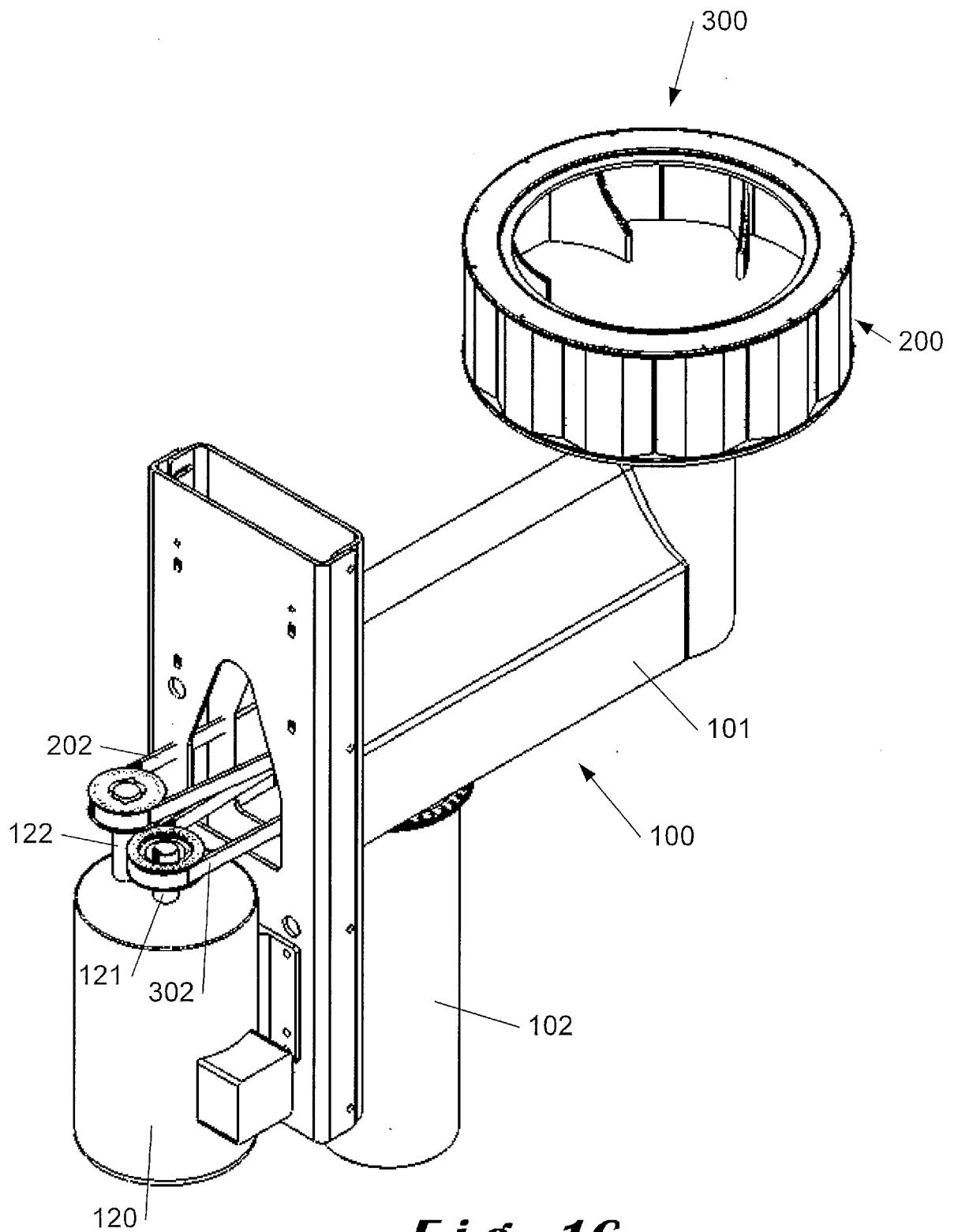


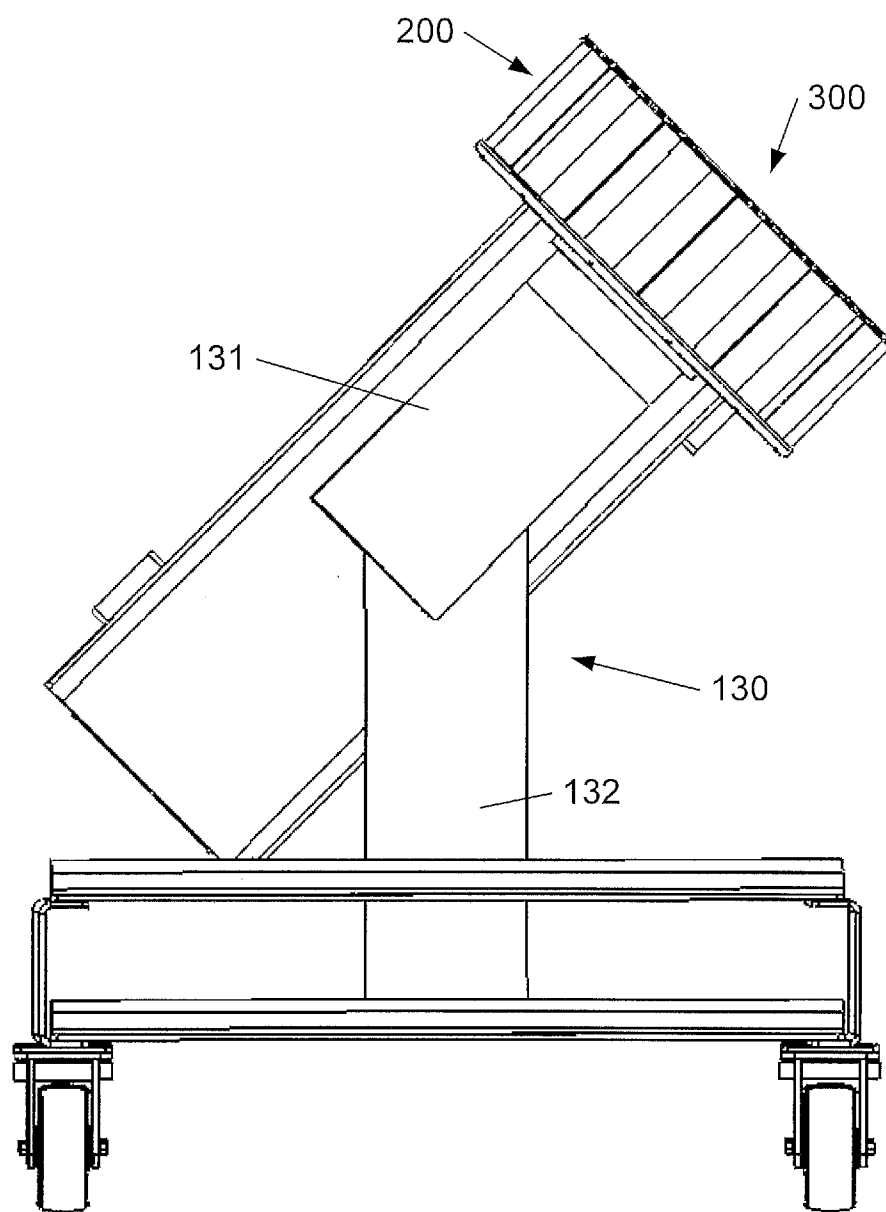
***Fig. 12***

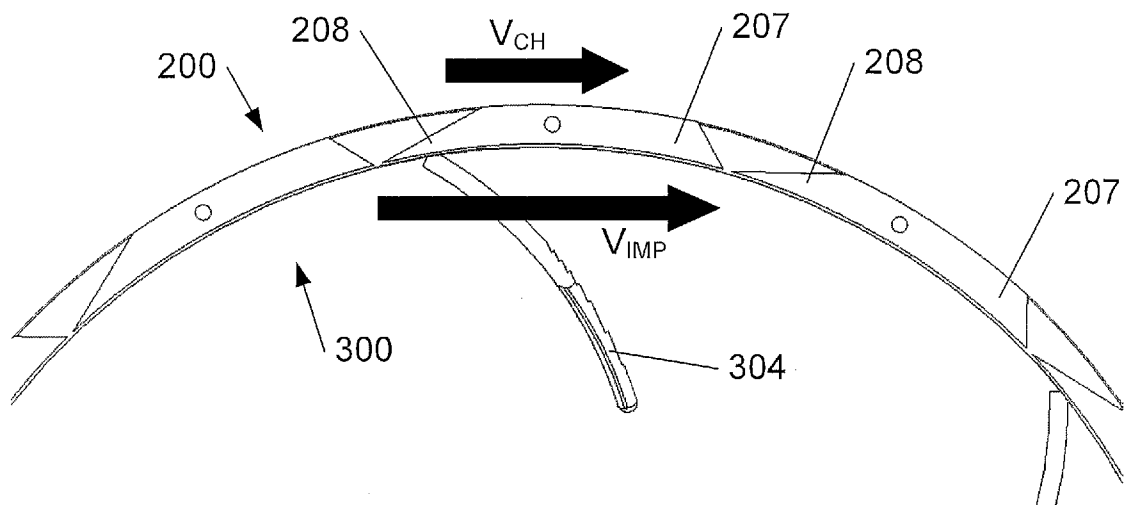
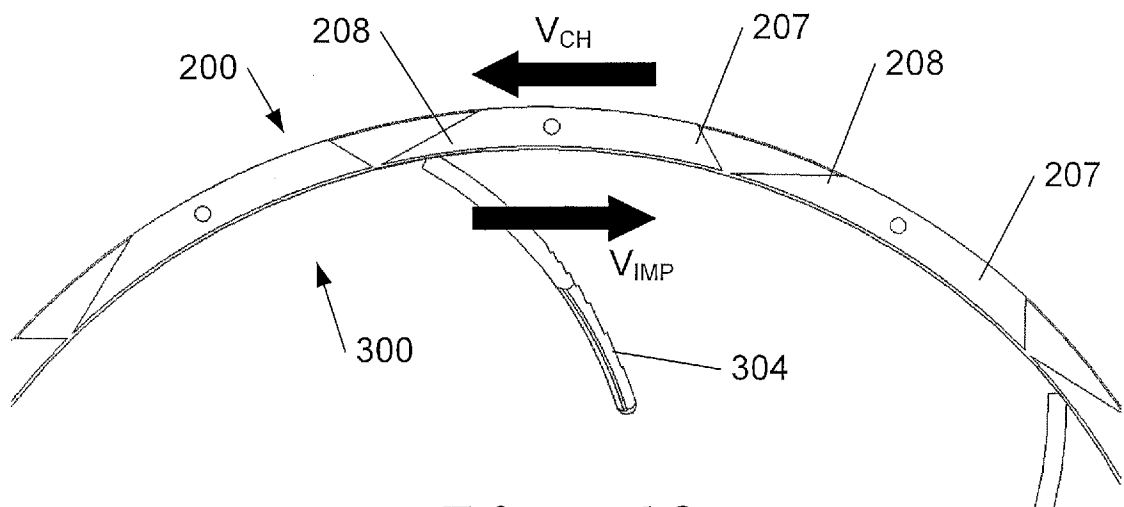


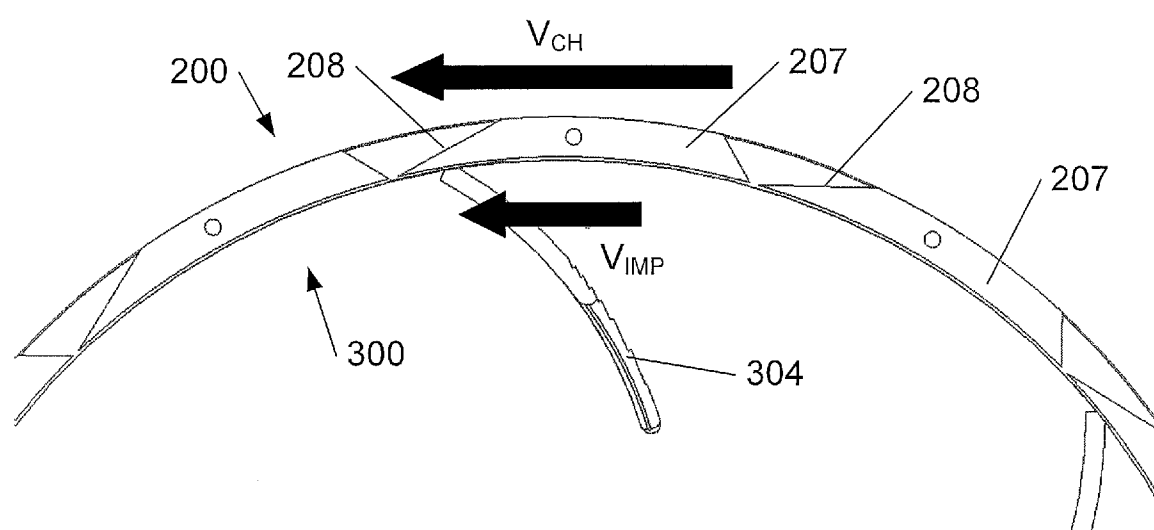
***Fig. 14***

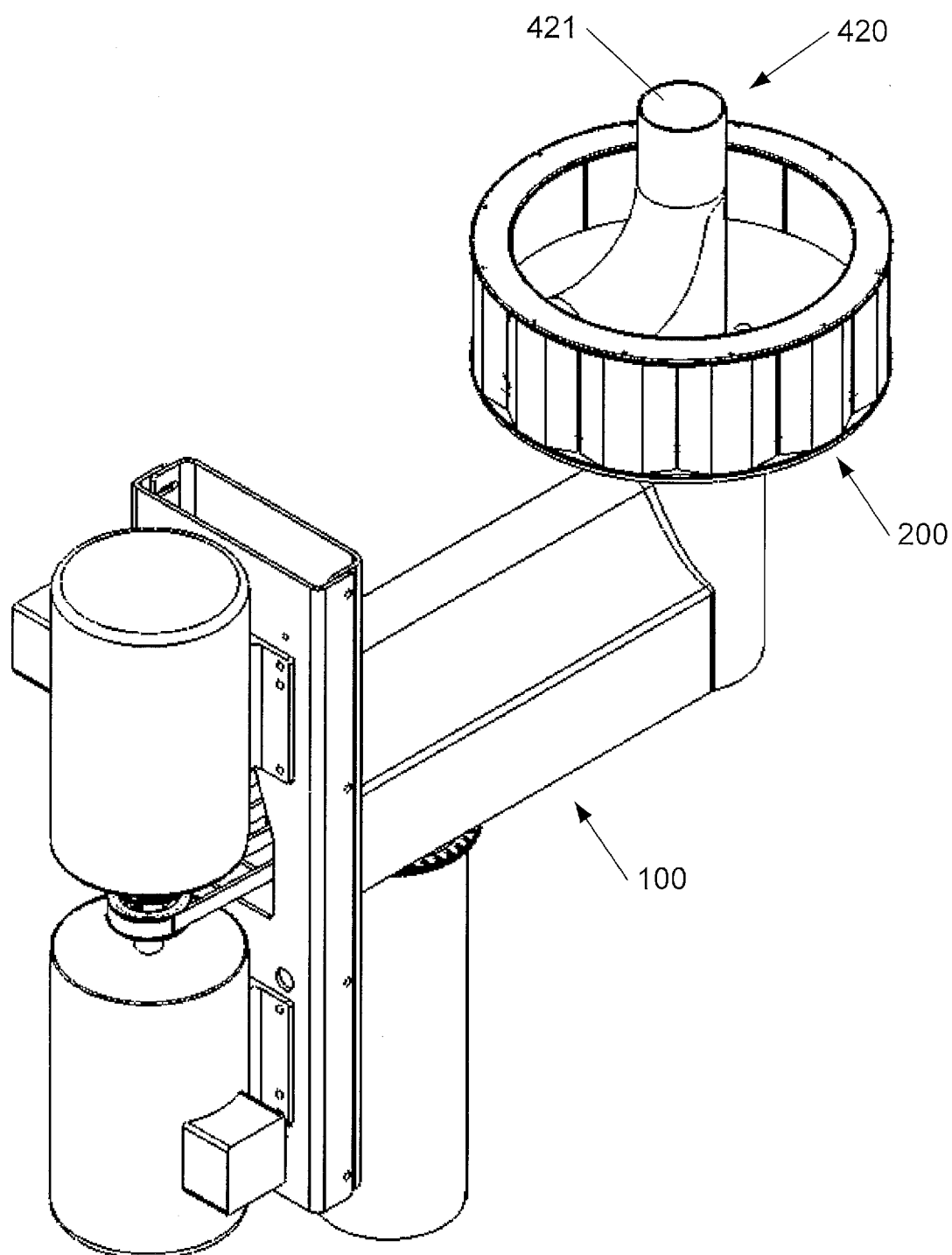
***Fig. 15***

***Fig. 16***

***Fig. 17***

***Fig. 18******Fig. 19***

***Fig. 20***

***Fig. 21***

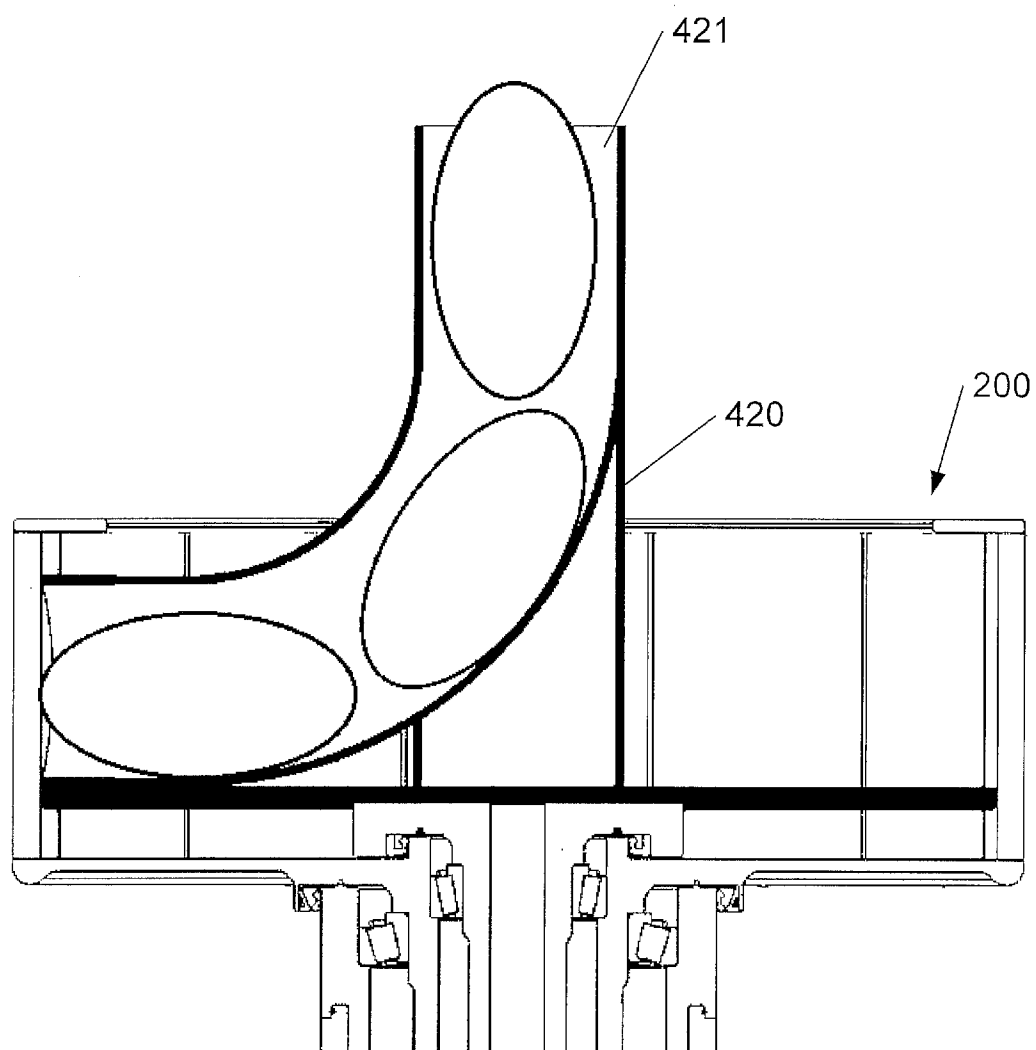
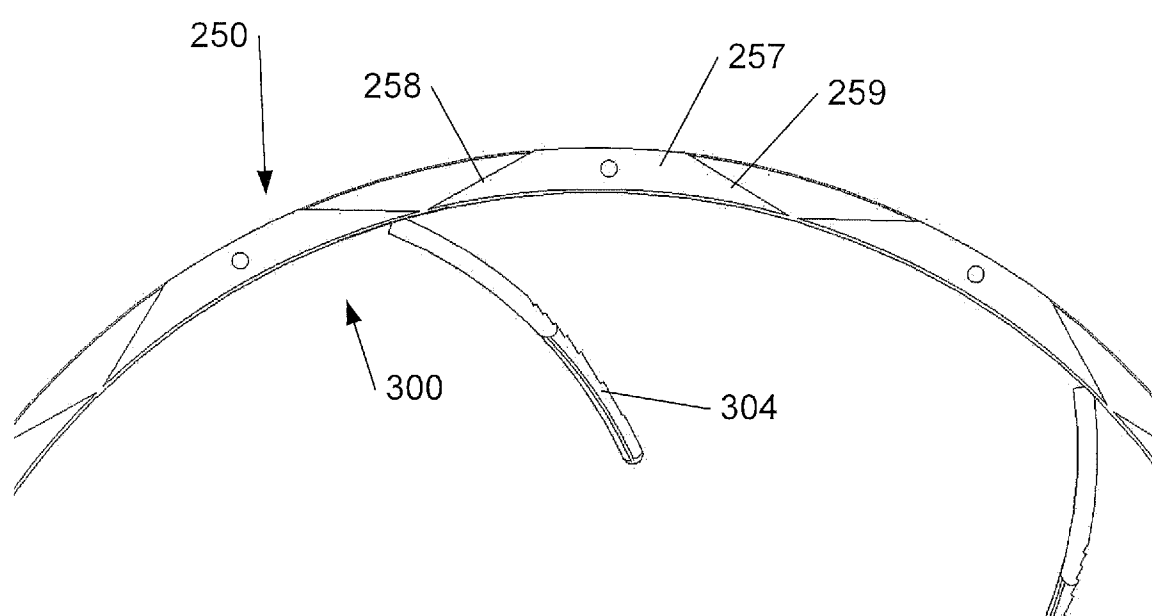


Fig. 22

***Fig. 23***

INTERNATIONAL SEARCH REPORT

International application No

PCT/EP2012/056401

A. CLASSIFICATION OF SUBJECT MATTER
INV. B26D7/06 B26D1/36
ADD.

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)
B26D

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)

EPO-Internal, WPI Data

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 4 604 925 A (WISDOM LAWRENCE W [US]) 12 August 1986 (1986-08-12)	1,2,4-7, 9,11-16, 18-23, 25-28, 30, 32-37, 39-41, 43-45
Y	column 2, line 12 - column 3, line 15; figures 1-2 ----- -/--	42



Further documents are listed in the continuation of Box C.



See patent family annex.

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Date of the actual completion of the international search

3 August 2012

Date of mailing of the international search report

14/08/2012

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Authorized officer

De Backer, Tom

INTERNATIONAL SEARCH REPORT

International application No

PCT/EP2012/056401

C(Continuation). DOCUMENTS CONSIDERED TO BE RELEVANT

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
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International application No

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