



(43) International Publication Date  
14 September 2017 (14.09.2017)

- (51) International Patent Classification:  
*F16L 33/24* (2006.01) *F16L 33/22* (2006.01)
- (21) International Application Number:  
PCT/EP2017/055295
- (22) International Filing Date:  
7 March 2017 (07.03.2017)
- (25) Filing Language: English
- (26) Publication Language: English
- (30) Priority Data:  
2016/0074 9 March 2016 (09.03.2016) IE
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- (81) Designated States (*unless otherwise indicated, for every kind of national protection available*): AE, AG, AL, AM, AO, AT, AU, AZ, BA, BB, BG, BH, BN, BR, BW, BY,

BZ, CA, CH, CL, CN, CO, CR, CU, CZ, DE, DJ, DK, DM, DO, DZ, EC, EE, EG, ES, FI, GB, GD, GE, GH, GM, GT, HN, HR, HU, ID, IL, IN, IR, IS, JP, KE, KG, KH, KN, KP, KR, KW, KZ, LA, LC, LK, LR, LS, LU, LY, MA, MD, ME, MG, MK, MN, MW, MX, MY, MZ, NA, NG, NI, NO, NZ, OM, PA, PE, PG, PH, PL, PT, QA, RO, RS, RU, RW, SA, SC, SD, SE, SG, SK, SL, SM, ST, SV, SY, TH, TJ, TM, TN, TR, TT, TZ, UA, UG, US, UZ, VC, VN, ZA, ZM, ZW.

- (84) Designated States (*unless otherwise indicated, for every kind of regional protection available*): ARIPO (BW, GH, GM, KE, LR, LS, MW, MZ, NA, RW, SD, SL, ST, SZ, TZ, UG, ZM, ZW), Eurasian (AM, AZ, BY, KG, KZ, RU, TJ, TM), European (AL, AT, BE, BG, CH, CY, CZ, DE, DK, EE, ES, FI, FR, GB, GR, HR, HU, IE, IS, IT, LT, LU, LV, MC, MK, MT, NL, NO, PL, PT, RO, RS, SE, SI, SK, SM, TR), OAPI (BF, BJ, CF, CG, CI, CM, GA, GN, GQ, GW, KM, ML, MR, NE, SN, TD, TG).

Published:

— with international search report (Art. 21(3))

(54) Title: A HIGH PRESSURE HOSE COUPLER

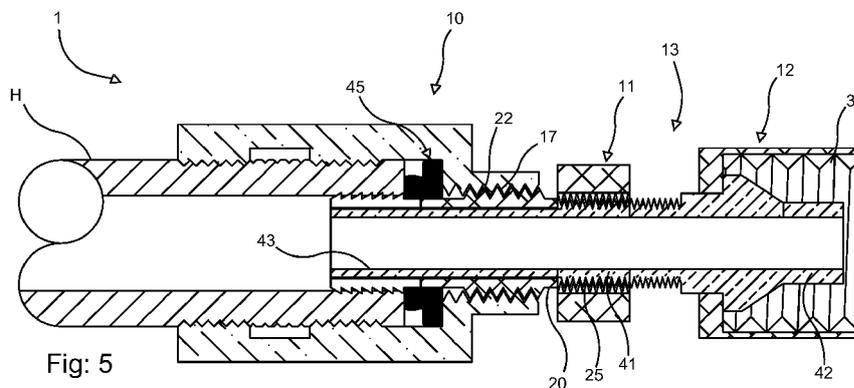


Fig: 5

(57) Abstract: A high pressure hose coupler (1) has a tubular connector (10) having internal threads (15, 16) at a distal end for engaging a hose (H) around the hose's external surface. These teeth grip into the hose's outer soft layer, and the hose is typically steel-reinforced. An internal collet (11) fits inside the hose (H), and engages the connector (10) at a proximal end to be retained in place. A hollow insert (13) has a distal end (40) configured to be inserted within the collet (11) to press the collet radially outwardly so that the hose wall is pressed between the connector and the collet. The insert (13) proximal end is configured to engage the collet at threads (41) on collet internal threads (25). The hose (H) is gripped very tightly by action of the insert (13) pressing against the tapered collet (11), in turn against the restraining force of the connector around the end of the hose.



“A High Pressure Hose Coupler”

INTRODUCTION

5 Field of the Invention

The invention relates to coupling of high pressure pipes or hoses, such as hydraulic hoses on heavy plant for construction, industrial, farming or marine applications.

10 At present, many such couplers require specialist equipment for installation, typically involving crimping with large forces. This is because of the perception that such techniques are required in order for the coupler to withstand the very high pressures involved, often of the order in excess of 500 bar. It can often be very inconvenient to await the services of a specialist to visit and make repairs to broken joints, and there can be considerable down-time of valuable plant. This  
15 can often happen out-of-hours during intensive farm work, for example.

It is known to provide a hose coupler with a tubular connector and a collet for insertion within a tube end, for example in US2009/0026760, EP0369834, and CN203082417.

20 The invention is directed towards providing an effective coupler for high pressure hoses, which can be manually applied without need for specialist equipment.

Summary of the Invention

According to the invention, there is provided a high pressure hose coupler comprising:

25 a tubular connector having internal threads at a distal end for engaging a hose around the hose's external surface,  
an internal collet having a distal end configured to fit inside a hose, and a proximal end to engage the connector to be retained in place, and  
a hollow insert having a distal tube configured to be inserted within the collet to press the  
30 collet radially outwardly so that the hose wall is pressed between the connector and the collet.

In one embodiment, the connector has at least two sets of internal threads configured for gripping a hose external surface, the sets being axially spaced apart by a gap to provide space for grit or grease.

5 In one embodiment, the collet has external threads and the connector has internal threads for inter-engagement. In one embodiment, the distal internal threads of the connector for engaging a hose and the proximal internal threads of the connector for engaging the collet are in the opposite sense.

10 In one embodiment, the connector has external flats for rotation by a spanner. Preferably, the connector includes a shoulder with an internal surface configured to seal axially against an end face of a hose end. In one embodiment, the coupler further comprises an annular resilient seal configured to fit within the tubular connector and form a seal against said shoulder internal surface. Preferably, the seal comprises a proximal flange and a distally-extending sleeve.

15 In one embodiment, the collet has external flats for rotation by a spanner. Preferably, the collet distal tube comprises ratchet teeth for gripping a hose internal surface.

In one embodiment, the collet has in the range of 2 to 10 ratchet teeth.

20 In one embodiment, the collet has longitudinal slits to facilitate resilience for radial pressing outwardly. Preferably, the collet has in the range of two to eight slits.

In one embodiment, the insert proximal end is configured to engage the collet at a proximal end  
25 of the collet. Preferably, the insert comprises external threads for engaging internal threads of the collet. In one embodiment, the insert tube had a uniform external surface at its distal end, without any taper.

In one embodiment, the insert tube internal bore has a tapered internal surface at its distal end to  
30 provide a funnel shape for assistance of hydraulic flow into the tube.

In one embodiment, the insert proximal end is configured to retain a nut for connection to a fixed coupler part. In one embodiment, the nut has internal threads for engaging a fixed coupler part around said proximal end of the insert.

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In one embodiment, the insert proximal end comprises a joint part of conical shape for engagement with a corresponding fixed conical joint part of a plant. Preferably, the coupler connector, collet, and insert are of metal material. Preferably, the metal is alloy steel.

5 In one embodiment, the insert has external flats for rotation by a spanner.

In another aspect, the invention provides a high pressure hose assembly comprising a hose and a coupler of any embodiment.

10 In a further aspect, the invention provides a heavy plant apparatus comprising a high pressure hose assembly of any embodiment.

#### Additional Statements

According to the invention, there is provided a high pressure hose coupler comprising:

15 a tubular connector having internal threads at a distal end for engaging a hose around the hose's external surface,  
an internal collet having a distal end configured to fit inside a hose, and a proximal end to engage the connector to be retained in place, and  
a hollow insert having a distal end configured to be inserted within the collet to press the  
20 collet radially outwardly so that the hose wall is pressed between the connector and the collet.

In one embodiment, the connector has at least two sets of internal threads configured for gripping a hose external surface, the sets being axially spaced apart by a gap to provide space for  
25 grit or grease.

In one embodiment, the collet has external threads and the connector has internal threads for inter-engagement.

30 In one embodiment, the distal internal threads of the connector for engaging a hose and the proximal internal threads of the connector for engaging the collet are in the opposite sense.

In one embodiment, the connector has external flats for rotation by a spanner.

In one embodiment, the connector includes a shoulder with an internal surface configured to seal axially against an end face of a hose end.

5 In one embodiment, the insert proximal end is configured to engage the collet at a proximal end of the collet.

In one embodiment, the insert proximal end is configured to retain a nut for connection to a fixed coupler part.

10 In one embodiment, the nut has internal threads for engaging a fixed coupler part around said proximal end of the insert.

In one embodiment, the insert proximal end comprises a joint part of conical shape for engagement with a corresponding fixed conical joint part of a plant.

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In one embodiment, the coupler connector, collet, and insert are of metal material.

In one embodiment, the metal is alloy steel.

20 In one embodiment, the collet has external flats for rotation by a spanner.

In one embodiment, the insert has external flats for rotation by a spanner.

In another aspect, the invention provides a high pressure hose assembly comprising a hose and a  
25 coupler of any embodiment.

## DETAILED DESCRIPTION OF THE INVENTION

### Brief Description of the Drawings

30 The invention will be more clearly understood from the following description of some embodiments thereof, given by way of example only with reference to the accompanying drawings in which:-

Fig. 1 is an exploded perspective view of a coupler of the invention;

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Figs. 2 and 3 are cross-sectional views showing a connector being applied around the end of a high pressure hose;

5 Fig. 4 is a cross-sectional view showing insertion of an internal collet within the end of the hose; and

Fig. 5 is a cross-sectional view showing also an insert within the collet.

## 10 Description of the Embodiments

A coupler 1 of the invention is suitable for coupling a high pressure hose H to a plant fitting without need for a crimping or other automated tool. A tubular connector 10 surrounds the end of the hose and engages it by threads. A collet 11 is placed inside the end of the hose, and an insert 13 is inserted within the collet to press it radially outwardly, so that the hose wall is  
15 sandwiched between the connector 10 and the collet 11 in a very tight grip to withstand the pressures of over 500 bar.

The ability to effectively couple and withstand these levels of hydraulic pressure are also contributed to by the particular manner of connecting the tubular connector 10 to the hose, how  
20 the collet is secured to the tubular connector by threads at a shoulder of the connector, and how the insert is secured to the collet and acts on the collet to radially press it out.

The tubular connector 10 has at its distal end axially separated internal threads 15 and 16 for engaging a hose H around the hose's external surface. These teeth grip into the hose's outer soft  
25 layer, and the hose H is typically steel-reinforced.

An internal collet 11 distal end fits inside the hose H within the volume encompassed by the connector 10, and has a proximal end which engages the connector 10 to be retained in place.

30 The hollow insert 13 has a distal end 40 configured to be inserted within the collet 11 to press the collet radially outwardly so that the hose wall is pressed between the connector and the collet. The insert 13 proximal end has external threads 41 configured to engage collet internal threads  
25.

The hose H is gripped very tightly by action of the insert 13 pressing against the tapered collet 11, in turn against the restraining force of the tubular connector around the end of the hose.

5 In more detail, the connector 10 has on its external surface flats 10(a) forming an hexagonal outer shape along that part which surrounds the end of the hose H, and a proximal shoulder 18. These allow the connector 10 to be conveniently rotated to engage a hose end, the threads 15 and 16 cutting into the hose external surface. The connector 10 distal end axially separated internal threads 15 and 16 are suitable for very tight and sealing gripping of a hose around the hose's outer surface. At the proximal end the connector 10 has internal threads 17 within the shoulder 10 18. Advantageously, the shoulder has an axially-facing face 18(a) facing in the distal direction, towards a hose end in use.

In one embodiment the part of the tubular connector between its distal end and the face 18(a) may have a tapered or frusto-conical internal shape narrowing in the proximal direction and ending at the face 18(a). The extent of this narrowing is small, for example 0.5mm to 2 mm of a reduction in internal diameter over a length of about 4cm for example.

There is a nut 12 for engagement of the coupler 1 with a fixed coupler component on a plant.

20 In one embodiment, there is also an annular axial resilient seal 45 of rubber material having a short sleeve portion 46 and a proximal flange 47. The internal diameter (ID) of the insert 13 is 10mm, and the outside diameter (OD) is 14mm, in one example. This is configured to engaging the connector 10 shoulder 18 end face 18(a) to seal between this face and the end face of a hose H.

25 The connector 10 distal threads 15 and 16 are left-handed. The axial gap between them is to provide a space 19 for any grit or grease in a manner in which it does not affect gripping of the connector onto the hose. This gripping is shown in Figs. 2 and 3. It can be done with a manual spanner engaging the flats 10(a).

30 The connector 10 proximal internal threads 17 are for receiving the internal collet 11, as shown most clearly in Fig. 4. The collet 11 has external threads 22 for engaging the connector proximal internal threads 17, allowing gradual axial movement of the collet 11 into the hose H. This movement of the collet 11 is facilitated by a bolt head 21 on a shank 20 at the proximal end of

the collet. At its distal end the collet 11 has tapered ratchet teeth 23 shaped for pushing in the distal direction into the hose, but to resist withdrawal. The collet 11 distal end has four equally spaced axial slits 24. At its proximal end the collet 11 has internal threads 25 within the head 21.

5 The insert 13 has a tube 40 with a bore 43 for flow of liquid from the hose H. The insert 13 also has a conical joint part 42 at the proximal end for engaging into a corresponding fixed joint part, for example of a heavy plant such as a digger. There are external threads 41 on the tube 40 for engaging the internal threads 25 of the collet 11. Advantageously, the tube 40 has a distal leading face which is at right angles to the curved surface, without any external shoulder or  
10 taper. Thus the circular rim corner is suited to pressing and retaining the collet radially outwardly. There may, however, be an internal taper at the distal end of the bore, to provide a funnel-shaped internal surface to facilitate flow of oil into the insert in use.

The nut 12 engages the insert 13, being retained by its head 42. It has hexagonal flats 30 and  
15 internal threads 31 for engaging onto a plant fixed coupler part.

In use, the coupler 1 is used to affix the end of a high pressure hose H to a fixed joint part on an item such as heavy plant. The sequence is as follows:

20 A) Insert the axial seal 45 into the connector 10 distal end and rotate the connector 10 around the end of a hose H as indicated by the arrow 50 in Fig. 2. First the threads 15 and then the threads 16 bite into the hose outer surface. Typically, the hose is of steel re-enforced rubber construction, with an outer texture amenable to tight gripping by the threads 15 and 16. Fig. 3 shows the connector 10 after being rotated into place. In this position an  
25 axial face 18(a) of the shoulder 18 of the connector 10 butts against the end face of the seal 45 flange 47, thereby contributing to the seal between the hose H and the connector 10. If the seal 45 is not used, the hose end face butts directly against the face 18(a).

If the connector has a narrowing internal diameter towards the end face 18(a) the hose is  
30 gripped ever more tightly during rotation to a greater extent than if it were of uniform internal diameter, and the extend of sealing is even greater.

B) As shown in Fig. 4, insert the collet 11 into the hose H end by rotation using the head 21. In this action the collet outer threads 22 engage the connector internal threads 17 in the

opposite sense to engagement of the connector threads 15 and 16 onto the hose H. The hose H is then gripped both internally and externally. However, the gripping force is typically insufficient.

- 5 C) As shown in Fig. 5, this grip is tightened by insertion of the insert 13, by rotation of the head 42 at its flats. In this action there is engagement of the insert outer threads 41 and the collet internal threads 25. This action is performed with a spanner so that the insert 13 distal tube 40 moves axially into the collet and thereby presses radially outwardly against the internal surface of the collet 11 in a rawl plug type of action to cause it to expand radially with the freedom of movement allowed by the four longitudinal slits 24. This causes the collet teeth 23 to grip more tightly into the internal fabric of the hose H.
- 10
- D) The proximal end of the insert 13 is then engaged with a fixed coupler part by pressing into a conical fixed plant component with the nut 12 internal threads 31 engaging external threads of the fixed plant component.
- 15

It will be appreciated that the invention provides for manual and very effective making of joints on high pressure equipment. There are sufficient engagement forces applied because of the internal and external gripping of the hose H, augmented by the insert pressing the collet radially outwardly. The gripping on the external surface of the hose H helps to ensure that grease or dirt does not affect seal integrity, any such contamination remaining in the gap 19 between the threads 15 and 16. This extent of coupling for high pressure hoses has been achieved despite the perception that specialist crimping equipment is required for coupling such hoses due to the high pressures.

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25 The invention is not limited to the embodiments described but may be varied in construction and detail. For example the coupler of the invention may be arranged to interconnect two hose ends, in which case it has a symmetrical configuration. The collet may have a number of slits in the range of 2 to 8 preferably, depending on the size of the hose to be coupled and hence of the coupling. Larger diameters will typically require more slits. While the collet is illustrated as having five ratchet teeth 23, it may have none or a number in the range of 1 to 10 for example, preferably two to eight. Also, it is not essential that an axial seal be inserted between the end face of the hose and the connector axial surface, although such a seal is typically advantageous because it helps to block a potential escape path for oil. Also, it is envisaged that the insert may

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be secured, for example by threads, directly to the connector instead of or in addition to the collet. While the connector internal and external threads have an opposite sense, this is not essential, and they could alternatively be in the same sense, and likewise the threads for engagement of the insert and the collet.

Claims

1. A high pressure hose coupler comprising:  
5 a tubular connector (10) having internal threads (15, 16) at a distal end for engaging a hose (H) around the hose's external surface,  
an internal collet (11) having a distal end (23, 24) configured to fit inside a hose, and a proximal end (22) to engage the connector to be retained in place, and  
a hollow insert (13) having a distal tube (40) configured to be inserted within the  
10 collet to press the collet radially outwardly so that the hose wall is pressed between the connector and the collet.
2. A high pressure hose coupler as claimed in claim 1, wherein the connector (10) has at least two sets of internal threads (15, 16) configured for gripping a hose external surface, the sets being axially spaced apart by a gap (19) to provide space for grit or grease.  
15
3. A high pressure hose coupler as claimed in claims 1 or 2, wherein the collet (11) has external threads (22) and the connector (10) has internal threads (17) for inter-engagement.
- 20 4. A high pressure hose coupler as claimed in claim 3, wherein the distal internal threads (15, 16) of the connector for engaging a hose and the proximal internal threads (17) of the connector for engaging the collet (11) are in the opposite sense.
- 25 5. A high pressure hose coupler as claimed in any preceding claim, wherein the connector has external flats (10(a)) for rotation by a spanner.
6. A high pressure hose coupler as claimed in any preceding claim, wherein the connector includes a shoulder (18) with an internal surface configured to seal axially against an end face of a hose end.
- 30 7. A high pressure hose coupler as claimed in claim 6, wherein the coupler further comprises an annular resilient seal configured to fit within the tubular connector and form a seal against said shoulder internal surface (18a).

8. A high pressure hose coupler as claimed in claim 7, wherein the seal comprises a proximal flange (47) and a distally-extending sleeve (46).
- 5 9. A high pressure hose coupler as claimed in any preceding claim, wherein the collet (11) has external flats (21) for rotation by a spanner.
10. A high pressure hose coupler as claimed in any preceding claim, wherein the collet (11) distal tube comprises ratchet teeth (23) for gripping a hose internal surface.
- 10 11. A high pressure hose coupler as claimed in claim 10, wherein the collet (11) has in the range of 2 to 10 ratchet teeth (23).
12. A high pressure hose coupler as claimed in any preceding claim, wherein the collet (11) has longitudinal slits (24) to facilitate resilience for radial pressing outwardly.
- 15 13. A high pressure hose coupler as claimed in claim 12, wherein the collet (11) has in the range of two to eight slits.
14. A high pressure hose coupler as claimed in any preceding claim, wherein the insert proximal end (41) is configured to engage the collet at a proximal end (25) of the collet.
- 20 14. A high pressure hose coupler as claimed in claim 14, wherein the insert (13) comprises external threads (41) for engaging internal threads (25) of the collet.
- 25 16. A high pressure hose coupler as claimed in any preceding claim, wherein the insert tube had a uniform external surface at its distal end, without any taper.
17. A high pressure hose coupler as claimed in any preceding claim, wherein the insert tube (40) internal bore (43) has a tapered internal surface at its distal end to provide a funnel shape for assistance of hydraulic flow into the tube.
- 30 17. A high pressure hose coupler as claimed in any preceding claim, wherein the insert tube (40) internal bore (43) has a tapered internal surface at its distal end to provide a funnel shape for assistance of hydraulic flow into the tube.
18. A high pressure hose coupler as claimed in any preceding claim, wherein the insert (13) proximal end (42) is configured to retain a nut (12) for connection to a fixed coupler part.
- 35 19. A high pressure hose coupler as claimed in claim 18, wherein the nut has internal threads (31) for engaging a fixed coupler part around said proximal end (42) of the insert.

20. A high pressure hose coupler as claimed in any preceding claim, wherein the insert proximal end comprises a joint part (42) of conical shape for engagement with a corresponding fixed conical joint part of a plant.
- 5 21. A high pressure hose coupler as claimed in any preceding claim, wherein the coupler connector, collet, and insert are of metal material.
22. A high pressure hose coupler as claimed in claim 21, wherein the metal is alloy steel.
- 10 23. A high pressure hose coupler as claimed in any preceding claim, wherein the insert has external flats (42) for rotation by a spanner.
24. A high pressure hose assembly comprising a hose and a coupler as claimed in any preceding claim.
- 15 25. A heavy plant apparatus comprising a high pressure hose assembly as claimed in claim 24.

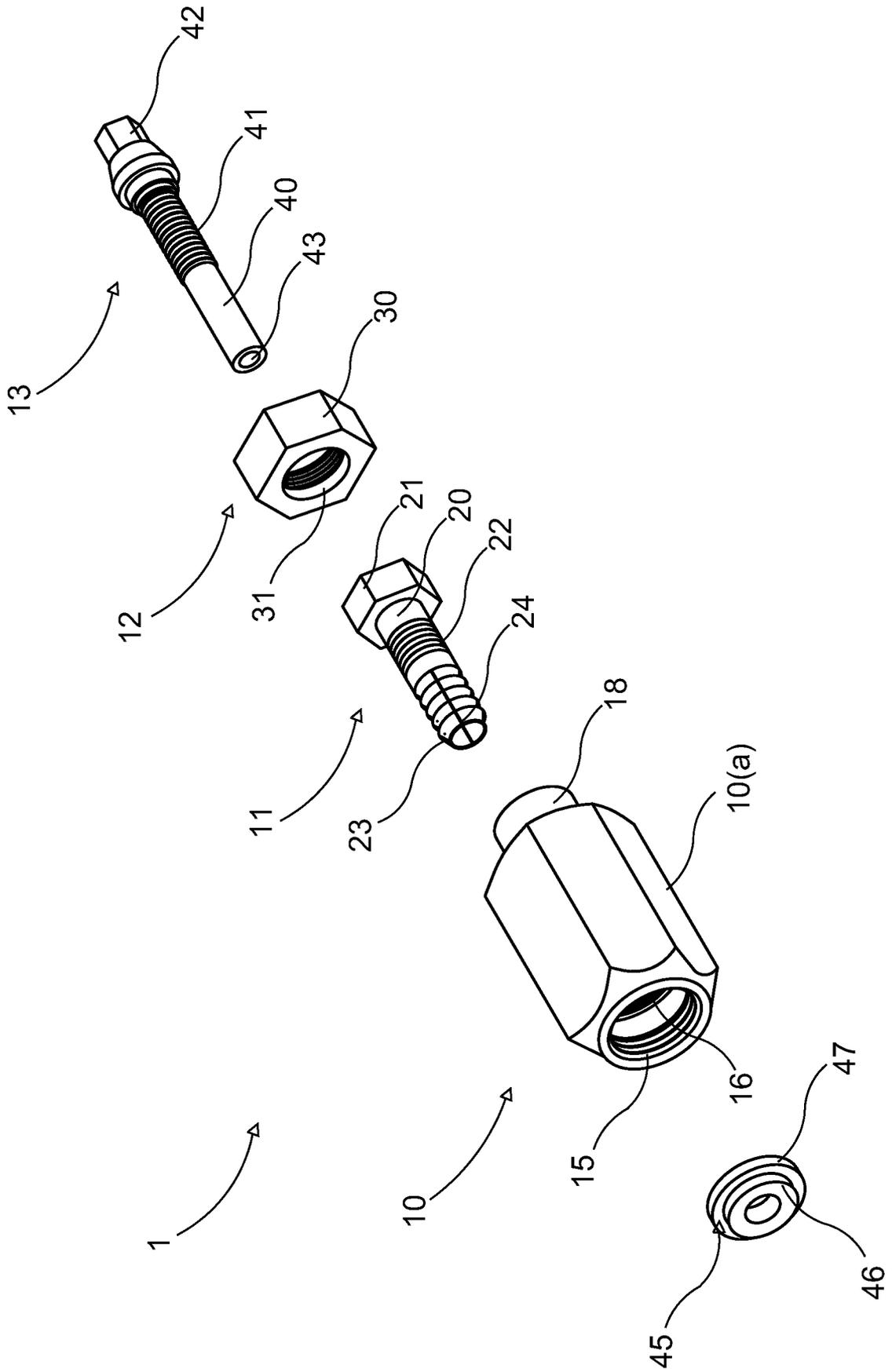


Fig: 1

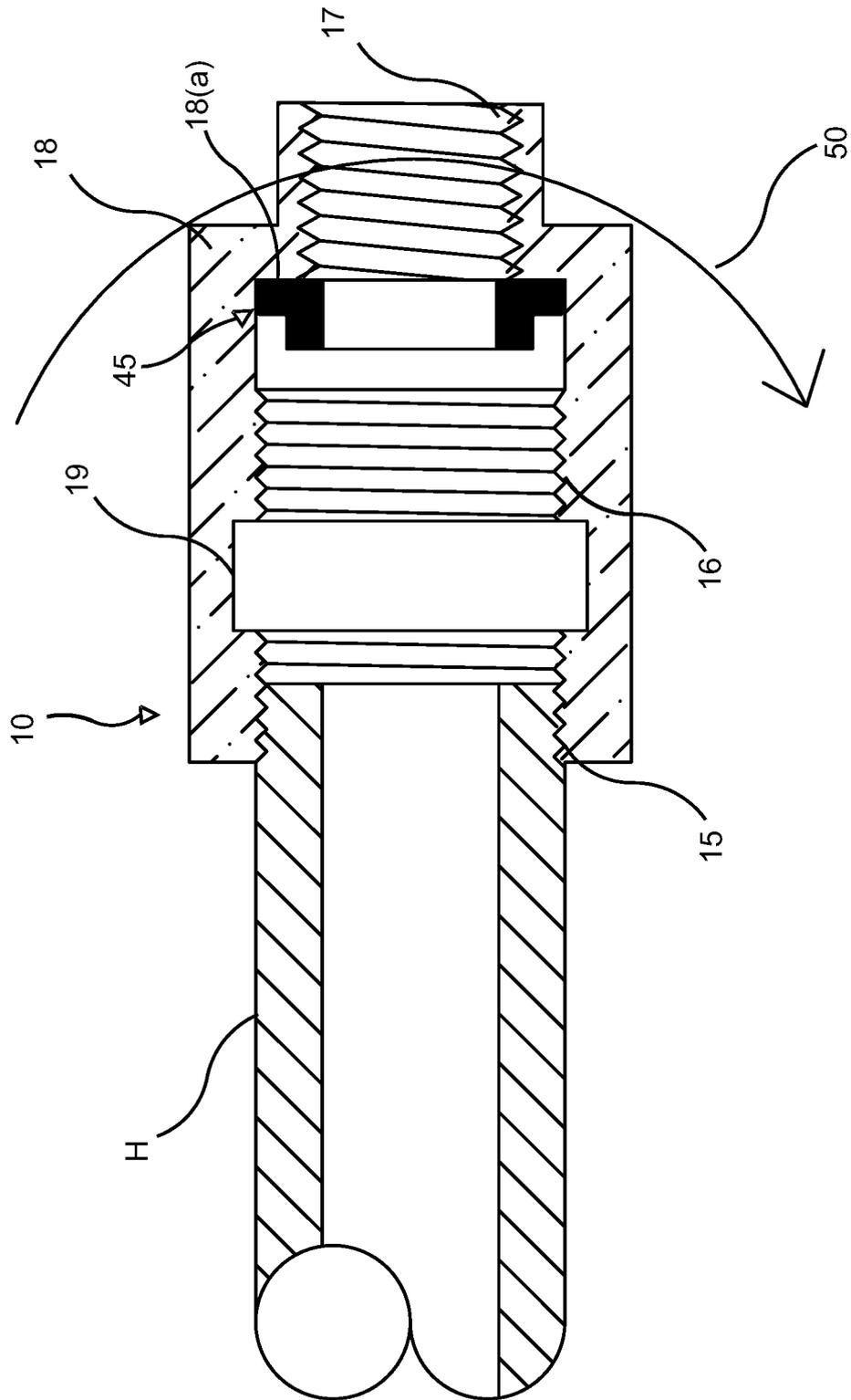


Fig: 2

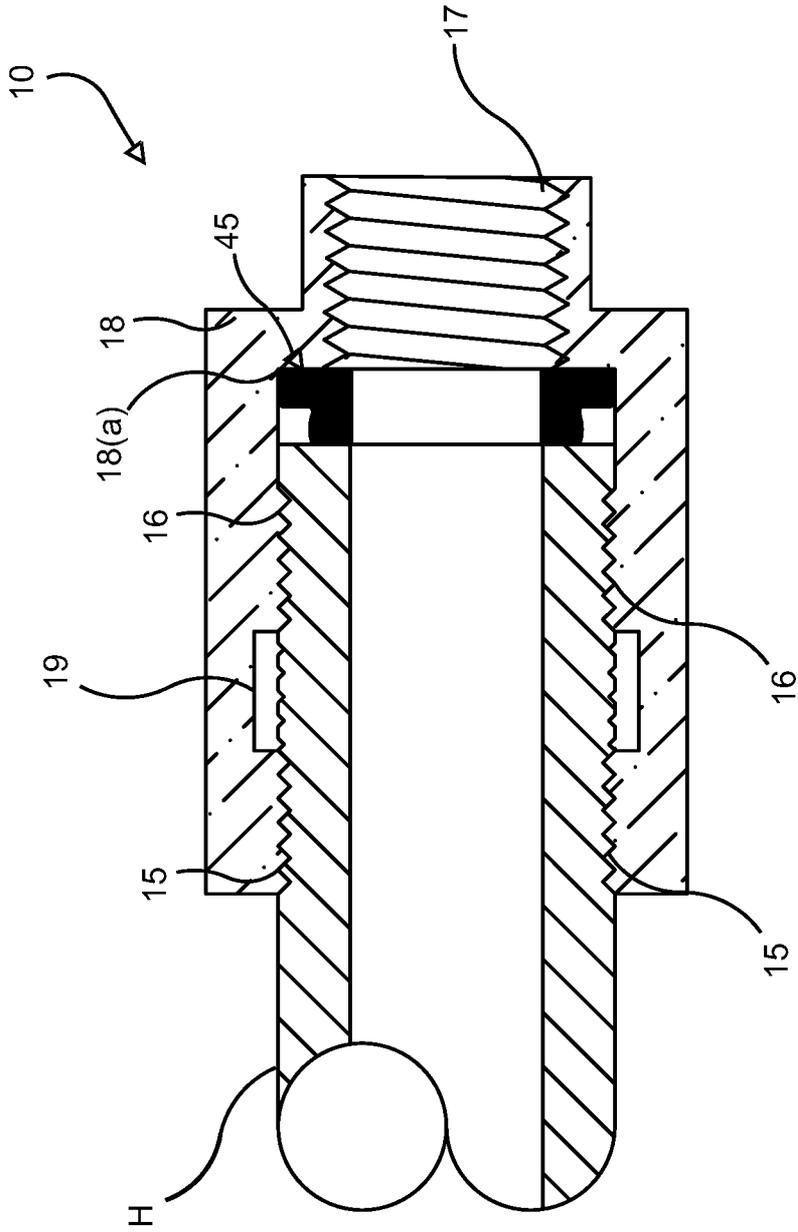


Fig: 3

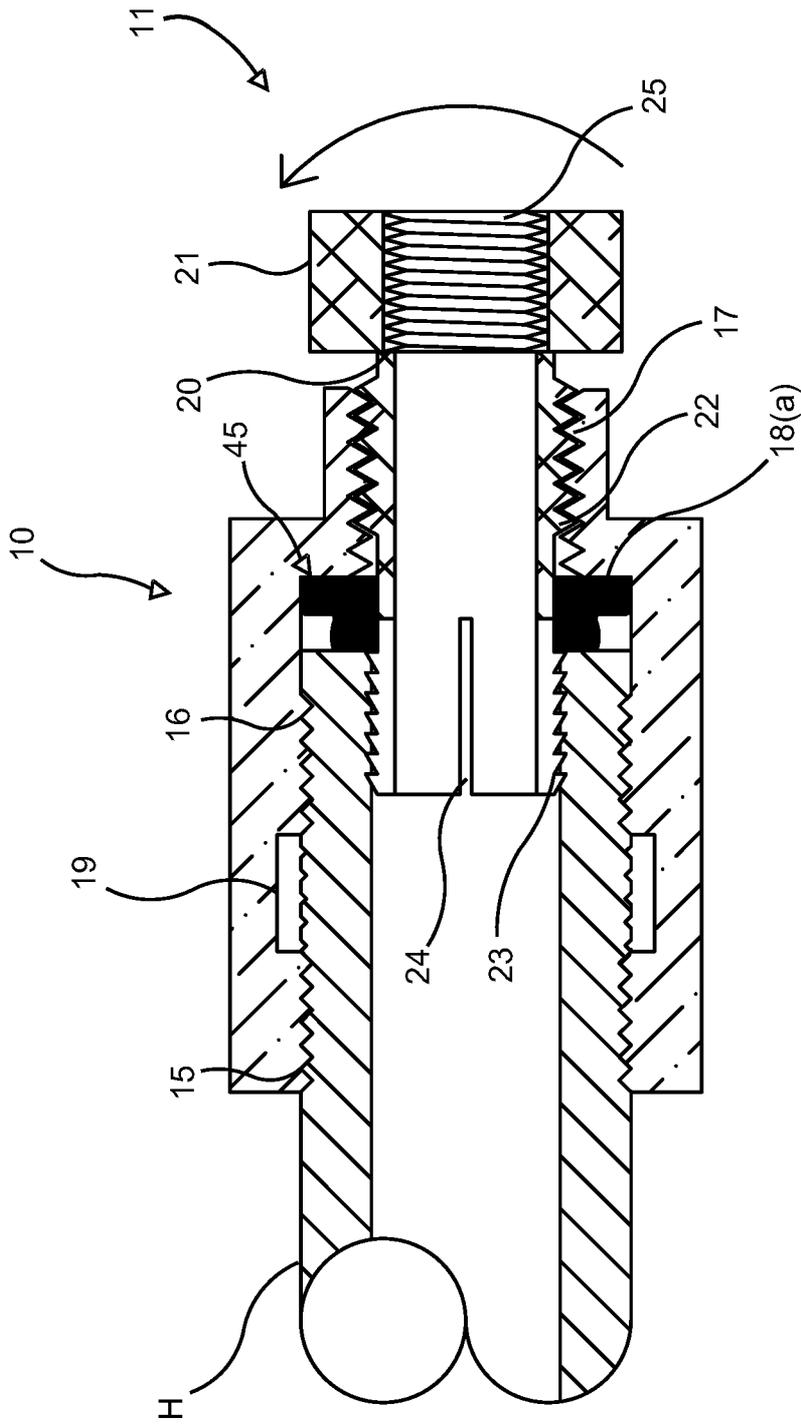


Fig: 4

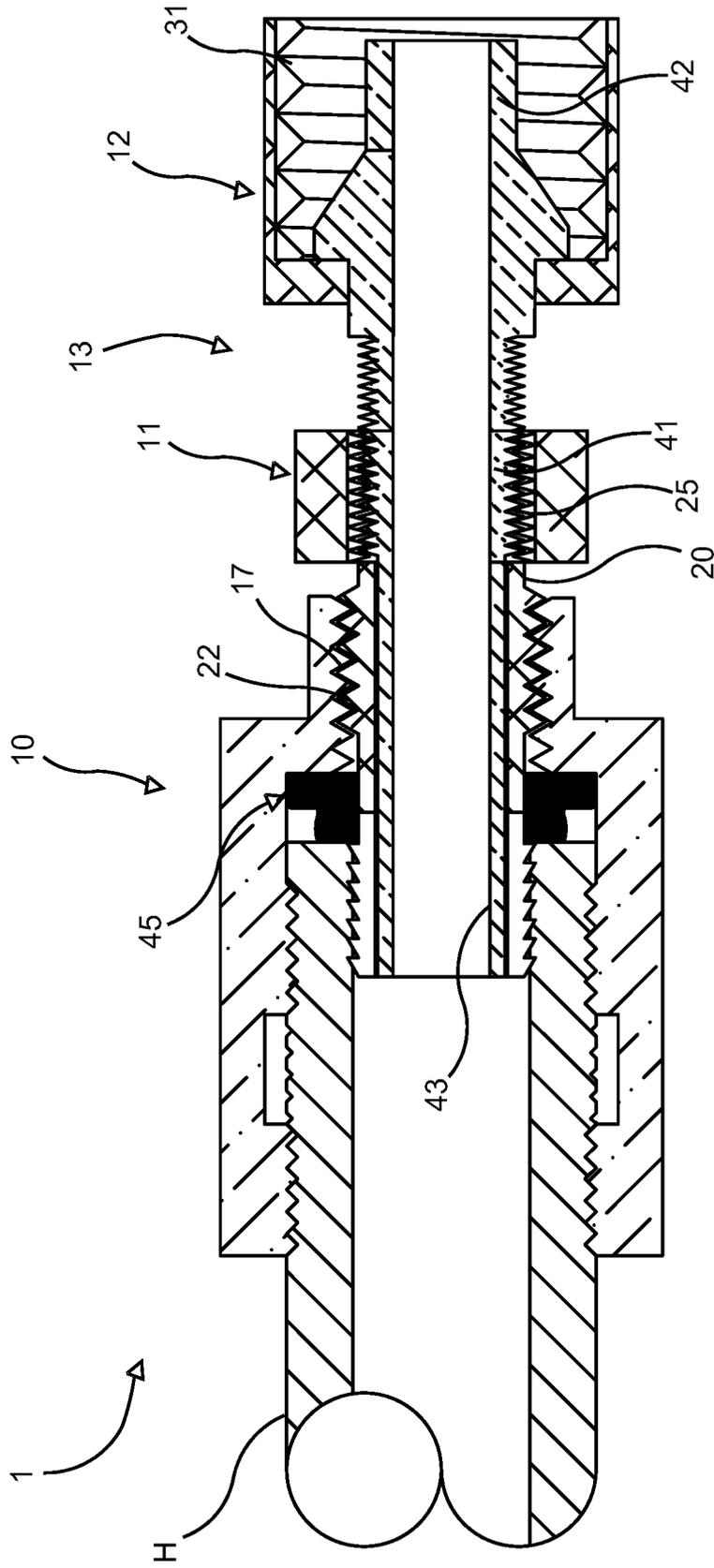


Fig: 5

**INTERNATIONAL SEARCH REPORT**

International application No  
PCT/EP2017/055295

**A. CLASSIFICATION OF SUBJECT MATTER**  
INV. F16L33/24 F16L33/22  
ADD.  
According to International Patent Classification (IPC) or to both national classification and IPC

**B. FIELDS SEARCHED**  
Minimum documentation searched (classification system followed by classification symbols)  
F16L

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practicable, search terms used)  
EPO-Internal, WPI Data

**C. DOCUMENTS CONSIDERED TO BE RELEVANT**

Category*	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	US 5 240 291 A (ZORNOW JEFFREY S [US]) 31 August 1993 (1993-08-31) column 3, lines 4-54; figures 1-3 -----	1-25
A	US 4 224 464 A (BUNNELL EDWARD D ET AL) 23 September 1980 (1980-09-23) column 1, line 51 - column 3, line 13; figures 1-5 -----	1-25

Further documents are listed in the continuation of Box C.

See patent family annex.

\* Special categories of cited documents :

<p>"A" document defining the general state of the art which is not considered to be of particular relevance</p> <p>"E" earlier application or patent but published on or after the international filing date</p> <p>"L" document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)</p> <p>"O" document referring to an oral disclosure, use, exhibition or other means</p> <p>"P" document published prior to the international filing date but later than the priority date claimed</p>	<p>"T" later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention</p> <p>"X" document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone</p> <p>"Y" document of particular relevance; the claimed invention cannot be considered to involve an inventive step when the document is combined with one or more other such documents, such combination being obvious to a person skilled in the art</p> <p>"&amp;" document member of the same patent family</p>
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Date of the actual completion of the international search <b>6 June 2017</b>	Date of mailing of the international search report <b>21/06/2017</b>
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Name and mailing address of the ISA/ European Patent Office, P.B. 5818 Patentlaan 2 NL - 2280 HV Rijswijk Tel. (+31-70) 340-2040, Fax: (+31-70) 340-3016	Authorized officer <b>Gutiérrez Royo, M</b>
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# INTERNATIONAL SEARCH REPORT

Information on patent family members

International application No

PCT/EP2017/055295

Patent document cited in search report	Publication date	Patent family member(s)	Publication date
US 5240291	A	31-08-1993	NONE
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US 4224464	A	23-09-1980	NONE
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