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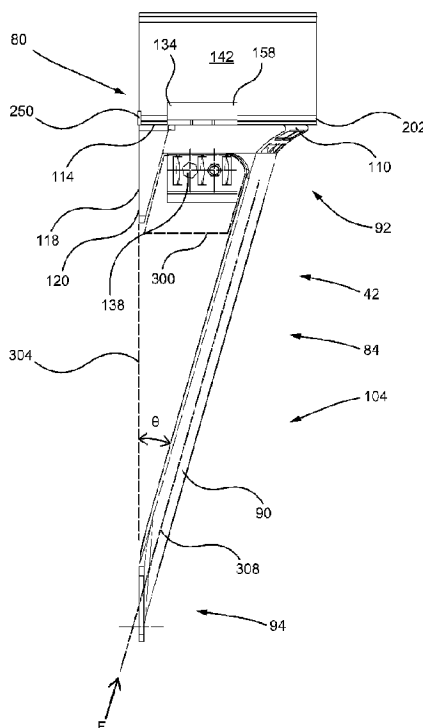
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(54) Title: STRUT PORTION WITH BUCKLING CAPABILITY FOR AERODYNAMIC SKIRT



(57) **Abrégé/Abstract:**

Aerodynamic skirt systems are used to reduce air drag of a vehicle. The present aerodynamic skirt system comprises a skirt panel, a plurality of struts, each strut comprising an upper portion for securing an upper portion of the skirt panel to the trailer, a strut member connected, at a first end thereof, to the upper portion of the strut, a lower portion for securing a lower portion of the skirt panel, the lower portion being connected to the strut member at a second end thereof, the strut member including a concave portion adapted to sustain a force applied by air routing on the aerodynamic skirt in an aerodynamic configuration thereof, the strut member being adapted to buckle and bend when a force exceeding a buckling threshold is applied on the aerodynamic skirt to move the aerodynamic skirt in an object avoidance configuration. A kit and a method of use thereof is also presented.



ABSTRACT

Aerodynamic skirt systems are used to reduce air drag of a vehicle. The present aerodynamic skirt system comprises a skirt panel, a plurality of struts, each strut comprising an upper portion for securing an upper portion of the skirt panel to the trailer, a strut member connected, at a first end thereof, to the upper portion of the strut, a lower portion for securing a lower portion of the skirt panel, the lower portion being connected to the strut member at a second end thereof, the strut member including a concave portion adapted to sustain a force applied by air routing on the aerodynamic skirt in an aerodynamic configuration thereof, the strut member being adapted to buckle and bend when a force exceeding a buckling threshold is applied on the aerodynamic skirt to move the aerodynamic skirt in an object avoidance configuration. A kit and a method of use thereof is also presented.

STRUT PORTION WITH BUCKLING CAPABILITY FOR AERODYNAMIC SKIRT

FIELD OF THE INVENTION

[0001] This invention generally relates to a strut for aerodynamic skirt. More precisely, the invention relates to a self-repositioning strut for securing an aerodynamic skirt to a vehicle to improve the aerodynamic efficiency of the vehicle.

BACKGROUND OF THE INVENTION

[0002] Road tractors are used to pull semi-trailers on roads to transport cargo. Aerodynamic apparatuses are installed on the road tractor and/or on the semi-trailer in order to reduce the aerodynamic air drag and improve fuel efficiency.

[0003] Trailer skirts made of rigid materials are installed on both lateral sides of a road trailer to help manage the flow of air around and underneath the trailer. Brackets, also made of rigid material, are affixed to the trailer to secure the skirts positioned thereto. These aerodynamic skirts are secured to the bottom portion of the trailer, or to the sides of the trailer's floor, to ensure proper positioning when the vehicle is moving.

[0004] People who are familiar with the trucking industry know that trailers are built in various configurations. Frame assembly of trailer can use members and beams of different dimensions. For example, an I-beam, that is commonly used in trailer manufacturing, also known as H-beam, W-beam (for "wide flange"), Universal Beam (UB), Rolled Steel Joist (RSJ), or double-T, is a beam with an I or H-shaped cross-section. The horizontal elements of the "I" are known as flanges, while the vertical element is termed the "web". I-beams are usually made of structural steel, or aluminum, and are used in construction and civil engineering. The web resists shear forces, while the flanges resist most of the bending moment experienced by the beam. Beam theory shows that the I-shaped section is a very efficient form for carrying both bending and shear loads in the plane of the web. An adaptable securing mechanism is hence desirable to adapt to a range of I-beam dimensions.

1 **[0005]** The skirts, because of their position under the trailer's floor and their
2 proximity with the road, are significantly vulnerable and might easily enter in contact
3 with surrounding obstacles. Portions of the securing mechanism holding the skirts,
4 when put under significant stress, plastically bend and/or break to effect the skirts'
5 position in respect to the road trailer thus reducing the efficiency of the skirts.
6 Additionally, the assembly can be crooked or not precisely aligned, which is causing
7 additional challenges to secure the aerodynamic skirt assembly to the vehicle.

8 **[0006]** Aerodynamic skirt assemblies in the art are complex to install given the
9 many adjustments required to match the precise configuration of each trailer. The
10 number of parts required to secure the strut to the trailer is generally significant and time
11 consuming to assemble. Also, the weight of the skirt assembly is important to prevent
12 unduly adding weight to the trailer and hence increase its fuel consumption.

13 **[0007]** Skirt supporting struts in the art have a linear deflection rate that is
14 proportional to the force applied thereto. The struts need to be rigid enough to remain in
15 their operating position and channel efficiently air around the trailer. This required
16 rigidity is significant and is detrimental to proper flexing of the skirt assembly when
17 contacting a foreign object.

18 **[0008]** Therefore, there exists a need in the art for an improved aerodynamic skirt
19 assembly over the existing art. There is a need in the art for an improved skirt-securing
20 strut adapted to recover its original shape after contacting a foreign object. There is also
21 a need for a skirt-securing strut that can flex, can be economically manufactured and
22 easily installed.

23 **SUMMARY OF THE INVENTION**

24 **[0009]** It is one aspect of the present invention to alleviate one or more of the
25 drawbacks of the background art by addressing one or more of the existing needs in the
26 art.

27 **[0010]** Accordingly, embodiments of this invention provide an improved trailer
28 aerodynamic skirt assembly over the prior art.

1 **[0011]** An aspect of one or more embodiments of the invention provides a shock-
2 resistant skirt assembly adapted to be installed on a semi-trailer to reduce the
3 aerodynamic drag produced by the movement of the trailer when pulled by a tractor.

4 **[0012]** An aspect of one or more embodiments of the invention provides a
5 concave strut for an aerodynamic skirt support mechanism to resiliently secure an
6 aerodynamic skirt panel on a trailer.

7 **[0013]** An aspect of one or more embodiments of the invention provides a self-
8 repositioning strut that is adapted to bend when it contacts a foreign object and recovers
9 its original position and shape thereafter.

10 **[0014]** An aspect of one or more embodiments of the invention provides a strut
11 adapted to bend/flex with significant displacement without breaking, when the skirt
12 assembly contacts a foreign object, and that is self-recovering its original position and
13 shape thereafter.

14 **[0015]** An aspect of one or more embodiments of the invention provides a one-
15 part integrated strut fixedly securing an upper portion and resiliently securing a lower
16 portion of a skirt panel.

17 **[0016]** An aspect of one or more embodiments of the invention provides a strut
18 portion having a height sized and designed to flex when the skirt assembly contacts a
19 foreign object and returns to equilibrium upon removal of the contact with the foreign
20 object.

21 **[0017]** An aspect of one or more embodiments of the invention provides a strut
22 including a flex portion adapted to keep an aerodynamic configuration when under a
23 predetermined mechanical load and adapted to buckle in an object avoidance
24 configuration when a load exceeding the predetermined mechanical load is applied
25 thereto.

26 **[0018]** An aspect of one or more embodiments of the invention provides a strut
27 including a flex portion with a concave portion.

1 **[0019]** An aspect of one or more embodiments of the invention provides a strut
2 portion including a semi-circular shape having a concave side facing the skirt panel.

3 **[0020]** An aspect of one or more embodiments of the invention provides a strut
4 portion including a semi-circular shape having a convex side facing the skirt panel.

5 **[0021]** An aspect of one or more embodiments of the invention provides a strut
6 portion including openings therein for receiving therein a connector portion therethrough
7 for securing the strut portion to a trailer.

8 **[0022]** An aspect of one or more embodiments of the invention provides a strut
9 portion including a stabilizer for transferring mechanical loads to a trailer when the strut
10 sustains a mechanical load.

11 **[0023]** An aspect of one or more embodiments of the invention provides a strut
12 portion including alignment guides for locating the strut portion to a trailer.

13 **[0024]** An aspect of one or more embodiments of the invention provides a strut
14 for securing an aerodynamic skirt panel to a trailer, the strut comprising an upper
15 portion for securing an upper portion of the aerodynamic skirt to the trailer, a strut
16 member connected, at a first end thereof, to the upper portion of the strut, a lower
17 portion for securing a lower portion of the aerodynamic skirt, the lower portion being
18 connected to the strut member at a second end thereof, the strut member including a
19 concave portion adapted to sustain a force applied by air routing on the aerodynamic
20 skirt in an aerodynamic configuration thereof, the strut member being adapted to buckle
21 and bend when a force exceeding a buckling threshold is applied on the aerodynamic
22 skirt to move the aerodynamic skirt in an object avoidance configuration.

23 **[0025]** An aspect of one or more embodiments of the invention provides an
24 aerodynamic skirt system comprising a skirt panel, a plurality of struts, each strut
25 comprising an upper portion for securing an upper portion of the skirt panel to the trailer,
26 a strut member connected, at a first end thereof, to the upper portion of the strut, a
27 lower portion for securing a lower portion of the skirt panel, the lower portion being
28 connected to the strut member at a second end thereof, the strut member including a

1 concave portion adapted to sustain a force applied by air routing on the aerodynamic
2 skirt in an aerodynamic configuration thereof, the strut member being adapted to buckle
3 and bend when a force exceeding a buckling threshold is applied on the aerodynamic
4 skirt to move the aerodynamic skirt in an object avoidance configuration.

5 **[0026]** Accordingly, embodiments of this invention provide an improved resilient
6 trailer aerodynamic skirt assembly over the prior art.

7 **[0027]** Other embodiments and further scope of applicability of the present
8 invention will become apparent from the detailed description given hereinafter.
9 However, it should be understood that the detailed description and specific examples,
10 while indicating preferred embodiments of the invention, are given by way of illustration
11 only, since various changes and modifications within the spirit and scope of the
12 invention will become apparent to those skilled in the art from this detailed description.

13 **[0028]** Additional and/or alternative advantages and salient features of the
14 invention will become apparent from the following detailed description, which, taken in
15 conjunction with the annexed drawings, disclose preferred embodiments of the
16 invention.

17 **BRIEF DESCRIPTION OF THE DRAWINGS**

18 **[0029]** Referring now to the drawings which form a part of this original disclosure:

19 **[0030]** Figure 1 is a side elevation view of a vehicle with a trailer in accordance
20 with at least one embodiment of the invention;

21 **[0031]** Figure 2 is a side elevation view of a portion of a trailer in accordance with
22 at least one embodiment of the invention;

23 **[0032]** Figure 3 is an isometric exploded view of an aerodynamic skirt in
24 accordance with at least one embodiment of the invention;

25 **[0033]** Figure 4 is an isometric view of a strut in accordance with at least one
26 embodiment of the invention;

1 **[0034]** Figure 5 is a front elevation view of a strut in accordance with at least one
2 embodiment of the invention;

3 **[0035]** Figure 6 is a back elevation view of a strut in accordance with at least one
4 embodiment of the invention;

5 **[0036]** Figure 7 is a side elevation view of a strut in accordance with at least one
6 embodiment of the invention;

7 **[0037]** Figure 8 is a top plan view of a strut in accordance with at least one
8 embodiment of the invention;

9 **[0038]** Figure 9 is an isometric view of a securing mechanism in accordance with
10 at least one embodiment of the invention;

11 **[0039]** Figure 10 is an isometric view of a securing mechanism in accordance
12 with at least one embodiment of the invention;

13 **[0040]** Figure 11 is a side elevation view of a securing mechanism in accordance
14 with at least one embodiment of the invention

15 **[0041]** Figure 12 is an isometric view of a portion of a securing mechanism in
16 accordance with at least one embodiment of the invention;

17 **[0042]** Figure 13 is a front elevation view of a portion of a securing mechanism in
18 accordance with at least one embodiment of the invention;

19 **[0043]** Figure 14 is a side elevation view of a securing mechanism in accordance
20 with at least one embodiment of the invention

21 **[0044]** Figure 15 is an isometric view of a strut attached to a trailer's floor beam in
22 accordance with at least one embodiment of the invention;

23 **[0045]** Figure 16 is a front elevation view of a strut attached to a trailer's floor
24 beam in accordance with at least one embodiment of the invention;

1 **[0046]** Figure 17 is a side elevation view of a strut attached to a trailer's floor
2 beam in accordance with at least one embodiment of the invention;

3 **[0047]** Figure 18 is a top plan view of a strut attached to a trailer's floor beam in
4 accordance with at least one embodiment of the invention;

5 **[0048]** Figure 19 is a front elevation view of a portion of a strut attached to a
6 trailer's floor beam in accordance with at least one embodiment of the invention;

7 **[0049]** Figure 20 is a front elevation view of a portion of a strut attached to a
8 trailer's floor beam in accordance with at least one embodiment of the invention;

9 **[0050]** Figure 21 is an isometric view of a portion of a strut attached to a trailer's
10 floor beam in accordance with at least one embodiment of the invention;

11 **[0051]** Figure 22 is a perspective view of a portion of a securing mechanism in
12 accordance with at least one embodiment of the invention;

13 **[0052]** Figure 23 is an isometric view of a strut attached to a trailer's floor beam in
14 accordance with at least one embodiment of the invention;

15 **[0053]** Figure 24 is a top plan view a strut attached to a trailer's floor beam in
16 accordance with at least one embodiment of the invention;

17 **[0054]** Figure 25 is an isometric view of a strut in accordance with at least one
18 embodiment of the invention;

19 **[0055]** Figure 26 is a front elevation view of a strut attached to a trailer's floor
20 beam in accordance with at least one embodiment of the invention;

21 **[0056]** Figure 27 is a back elevation view of a two struts attached to a trailer's
22 floor beam having two different heights in accordance with at least one embodiment of
23 the invention;

24 **[0057]** Figure 28 is a back elevation view of a strut in accordance with at least
25 one embodiment of the invention;

1 **[0058]** Figure 29 is a side elevation view of a strut in accordance with at least one
2 embodiment of the invention;

3 **[0059]** Figure 30 is a top plan view of a strut in accordance with at least one
4 embodiment of the invention;

5 **[0060]** Figure 31 is a perspective view of a portion of a strut in accordance with at
6 least one embodiment of the invention;

7 **[0061]** Figure 32 is a perspective view of a portion of a strut in accordance with at
8 least one embodiment of the invention;

9 **[0062]** Figure 33 is a perspective view of a portion of a strut in accordance with at
10 least one embodiment of the invention;

11 **[0063]** Figure 34 is a perspective view of a portion of a strut in accordance with at
12 least one embodiment of the invention;

13 **[0064]** Figure 35 is a perspective view of a portion of a strut in accordance with at
14 least one embodiment of the invention;

15 **[0065]** Figure 36 is a perspective view of a portion of a strut in accordance with at
16 least one embodiment of the invention;

17 **[0066]** Figure 37 is a top plan section view of a portion of a strut in accordance
18 with at least one embodiment of the invention;

19 **[0067]** Figure 38 is a side elevation view of a strut in accordance with at least one
20 embodiment of the invention;

21 **[0068]** Figure 39 is a graph illustrating a load vs displacement of a strut in
22 accordance with at least one embodiment of the invention;

23 **[0069]** Figure 40 is a perspective view of a strut in accordance with at least one
24 embodiment of the invention; and

1 **[0070]** Figure 41 is a perspective view of a strut in accordance with at least one
2 embodiment of the invention.

3 **DETAILED DESCRIPTION OF PREFERRED EMBODIMENTS**

4 **[0071]** A preferred embodiment of the present invention is described below with
5 reference to the drawings. Figure 1, Figure 2 and Figure 3 illustrate a road tractor 10
6 with a trailer 20 attached thereto equipped with a pair of skirt assemblies 30, installed
7 on each side of the trailer 20 along a longitudinal axis 34, adapted to deflect and direct
8 the airflow around the trailer 20. Each aerodynamic skirt assembly 30 includes a skirt
9 panel 38, adapted to be disposed on the side of the trailer 20, and a plurality of securing
10 mechanisms 42 adapted to secure the skirt panel 38 to the trailer 20. The securing
11 mechanisms 42 are visible although not clearly illustrated in Figure 1, Figure 2 and
12 Figure 3 and will be discussed in more details below. Once installed on the trailer 20,
13 the skirt assembly 30 helps channel the flow of air around the trailer 20 to reduce the air
14 drag of the vehicle when the trailer 20 moves on the road, pulled by the road tractor 10.
15 One can appreciate from Figure 1 that the trailer 20 includes slider rail 24 used to adjust
16 the location of the set of wheels 50 on the trailer 20, thus interacting with the length of
17 the skirt panel 38.

18 **[0072]** The skirt assembly 30 of the present embodiment is mostly located under
19 the trailer 20, between the wheels 46 of the road tractor 10 and the wheels 50 of the
20 trailer 20. The skirt panels 38 can alternatively extend forward up to the trailer supports
21 54, also known as landing gears, and be secured thereto thus preventing complex skirt
22 panel 38 arrangements through the securing mechanism 42. The skirt panels 38 are
23 substantially vertically positioned on each side of the trailer 20, preferably located as a
24 vertical continuity of the trailer's side/wall, with a clearance with the ground by
25 illustratively about 15-25 centimeters (about 6 to 12 inches). The air management
26 around the trailer 20 provided by the skirt assembly 30 reduces the air drag created by
27 the trailer 20 by directing the flow of air around the trailer 20. The flow of air would
28 otherwise turbulently move around and below the trailer 20 to create substantial air
29 drag. The airflow management around the trailer 20 provided by the skirt assembly 30

1 helps maintain laminar airflow around the trailer 20 that diminishes fuel consumption of
2 the road tractor 10. The skirt assembly 30 also improves the safety of the vehicle by
3 providing a barrier that can substantially restrict objects to get under the trailer 20.

4 **[0073]** As illustrated, the skirt panel 38 is shaped with an optional progressive
5 height 62 from a front portion 58 thereof. The skirt panels 38 can alternatively also be
6 installed at an angle, in respect to the vertical (not illustrated), on the trailer 20 to
7 change the airflow pattern around the trailer 20 and more precisely adjust the
8 aerodynamics to a specific vehicle shape.

9 **[0074]** It can be appreciated in Figure 2 that each skirt panel 38 is installed
10 directly on the side of the trailer 20 and, when seen from above, the front portion 58 that
11 optionally progressively proximally leans angle 66 toward the center 34 of the trailer 20.
12 The recessed front portion 58 of the skirt panel 38 improves the management of
13 turbulent airflow generated by the road tractor 10 thus improving the aerodynamic
14 efficiency of the skirt assembly 30. Additional explanation about the shape of the skirt
15 panel 38 will be provided in further details below.

16 **[0075]** Figure 4 throughout Figure 8 are illustrating a securing mechanism 42
17 including a connector portion 80 and a strut portion 84 both centered along vertical axis
18 44. The strut portion 84 includes a strut member 90, with a flex portion 104 thereof,
19 securing the skirt panel 38 to a lower portion 94 thereof. In the illustrated embodiment, a
20 planar section 98 includes holes 102 therein for securing the lower portion 94 of the skirt
21 panel 38 with fasteners or rivets, for example. The strut portion 84 includes a securing
22 portion 106 to which is connected the strut portion 84. The securing portion 106
23 includes a stabilizer 110, a trailer contacting portion 114, a skirt panel contacting portion
24 118 and a pair of securing wings 120. A stabilizer 110 is located proximal to the
25 longitudinal axis 34 of the trailer 20 when installed on a trailer 20, proximally extending
26 from a projection 70 of the strut member 90 to create a lever that help sustains the
27 loads applied on the strut portion 84 by the skirt panel 38. The skirt panel contacting
28 portion 118 includes a planar section 122 provided with a series of holes 126 therein for
29 securing an upper portion 92 of the skirt panel 38 with fasteners or rivets, for example.

Both planar section 98, 122 are preferably aligned along a unique vertical plane 74 for efficiently contacting the skirt panel 38. The trailer contacting portion 114 includes openings 130 therein to receive therein the connector portion 80 for securing the strut portion 84 to the trailer 20.

[0076] The connector portion 80 is embodied as two opposed clamps 134 configured to be secured together with, for instance, two fasteners 138. The two opposed clamps 134 are securing together the trailer contacting portion 114 of the strut portion 84 with a lower portion of an I-beam as illustrated in Figure 13 throughout Figure 18. Each clamp 134 uses an inclined member 160 and two contacting portions 162, 166 for securing the connector portion 80 with the I-beam 142. However, we will first describe in further details the connector portion 80 before discussing the interaction between the strut portion 84, the connector portion 80 and the I-beam 142.

[0077] Moving now to Figure 10 throughout Figure 11, a connector portion 80 is illustrated. The connector portion 80 is preferably made of two similar opposed clamps 134 to simplify the assembly and reduce the manufacturing cost. The clamps 134 are made of a strong material capable of withstanding significant mechanical loads and can be shaped with a punch and die process. Metal, or the like, can be used. A material resistant to corrosion, or a protective layer, is also desirable given the condition of use of the connector portion 80 to prevent rust to undesirably attack the connector portion 80.

[0078] Each clamp 134 is, in the present embodiment, made of bent sheet metal that is forming a collecting portion 146, a securing portion 150 and a contacting portion 154. The collecting portion 146 includes a collecting member 158 embodied as an inclined member 160 positioned at an angle α , that is about between 30 degree and about 45 degree, and more preferably about 34 degree in the illustrated embodiment, to collect therein sections of I-beams 142 having different dimensions and thicknesses. The collecting member 158 merges into a first contacting portion 162, configured to laterally abut sides of an I-beam 142, that merges into a second contacting portion 166 configured to vertically contact a surface of the connector portion 80. A securing portion

170 is formed between a first protruding member 174 and a second protruding member 178. Both members 174, 178 are extending about a similar distance to allow leveled abutment of the two clamps 134 when secured and pulled toward each other with an intervening I-beam 142 having optimal dimensions. The second protruding member 178 is also helpful to prevent premature rotation of the assembly when tightening the bolts 138 by its lever action and touching each other along a line to provide resistance to rotation to the assembly. Additional bents 182 are performed on the clamp 134 to increase the mechanical strength thereof. An elastic member 186, embodied as an elastic band in the embodiment, is used to keep both clamps 134 together in clamping position prior to secure the clamps 134 to the I-beam 142. Helical springs 188 can optionally be used to pretense the assembly as illustrated in Figure 10. The pre-assembly of the two clamps 134 are allowing suspending the pre-assembly of the two clamps 134 to the I-beam 1 to connect the parts of the aerodynamic skirt assembly 30 in place before securing them in a final operating configuration. Distance 190 is preferably configured to be close to zero (0) when the two clamps are secured toward each other and two times distance 194 is preferably configured to be close to a thickness of a central member of the I-beam 142 when the clamps 134 are secured to a narrow I-beam 142. Conversely, distance 190.1 and 190.2 is going to increase when the clamps 134 are secured to a wider I-beam. Holes 198 are made in the securing portion 170 to accommodate bolts 138 therein to secure both clamps 134 together. The interaction of the securing mechanism 42 with an I-beam 142 is depicted in Figure 15 throughout Figure 16.

[0079] It can be appreciated from Figure 13 throughout Figure 18, the collecting portion 146, with the inclined member 160, is compressing together vertically (y) and laterally (x) the trailer-contacting portion 114 and a lower portion 202 of the I-beam 142. The illustrated structure also has the capacity to adapt to a variety of I-beams 142. The lower portion 202 (bottom flange) of the I-beam can be wider 206 and/or thicker 210 and still be captured and secured by the connection portion 90 of the securing mechanism 42. Typically, the lower portion 202 of the I-beam 142 is illustratively varying from about 38 mm to about 80 mm width, and thickness of about 3 mm to about 13 mm. This is illustrated in Figure 19 and in Figure 20.

1 **[0080]** As identified in Figure 17, the strut portion 84 is, for instance, sized and
2 designed to buckle when a force F of about between 400 N to 600 N is applied along
3 the longitudinal axis 308. The buckling force F can alternatively be between about 450
4 N to 550 N. A narrower range of force F can alternatively be between about 470 N to
5 520 N without departing from the scope of the present invention. Corresponding force
6 applied normal to the skirt panel can be inferred from the axial buckling force above.

7 **[0081]** One can appreciate the I-beam 142 is parallel with the central axis 144 of
8 the I-beam 142 in Figure 15 throughout Figure 21 and in Figure 23 and Figure 24. The
9 securing mechanism 42 is constructed to keep the pair of clamps 134 aligned and allow
10 misalignment of the strut portion 84 in respect with the I-beam 142 of the trailer 42. The
11 alignment between the clamps 134 can be made by contacting the contacting portions
12 154 together. However, in a case of a wider I-beam 142, a spacer 220 can be required
13 to space apart the contacting portions 154. The spacer 220, better seen in Figure 22
14 can be made of different material having suitable mechanical properties like steel,
15 aluminum or plastic. The spacer 220 includes optional voids 224 and ribs 228 to lighten
16 the spacer 220, reduce curing time, eliminate sink marks and reinforce desirable areas
17 thereof. The spacer further includes fastener-receiving portions 228 configured to
18 receive fasteners 138 therein. A misalignment between the strut portion 84 and the I-
19 beam 142 of the trailer 42 can occur if the trailer 20 has I-beams 142 that are not
20 perfectly aligned and, depending of the installation configuration of the skirt panel 38
21 along the trailer 20, the front portion 58 of the skirt panel 38 can proximally lean toward
22 the center of the trailer 20 hence progressively curving or bending and be located at an
23 angle with the I-beam 142 of the trailer 20 as illustrated in Figure 23 and Figure 24. The
24 size of the openings 130 is designed to accommodate a flexible installation of the pair of
25 clamps 134 that can be located to accommodate position variation with, for instance, an
26 angle β thereof.

27 **[0082]** As mentioned above, the strut portion 84 is preferably located on the
28 trailer 20 to have the exterior of the skirt panel 38 vertically aligned, flush with the side
29 wall of the trailer 20. The skirt panel 38 is hence used as a vertical continuity of the
30 trailer's 20 side wall. The alignment of the strut portion 84 might be challenging to

1 achieve. Alignment guides 250 are provided on the trailer-contacting portion 114 to
2 assist the positioning and the installation of the strut portion 84 on the trailer 20. The
3 trailer-contacting portion 114 incorporates alignment guides 250, embodied as
4 protruding members, located at predetermined locations on the trailer-contacting portion
5 114 from the skirt panel contacting portion 118 used as reference plane for laterally
6 locating the skirt panel 38. The alignment guides 250 can be seen throughout the
7 Figures using some different configurations of distance and shape. The alignment
8 guides 250 can alternatively be embodied as extrusions molded in the strut portion 84,
9 one or many series of holes for receiving added pins, metallic or not, removable stems,
10 clips or the like without departing from the scope of the present description. One way to
11 facilitate the installation of the securing mechanism 42 is to assemble the connection
12 portion 80 to the strut portion 84 prior to assemble the connection portion 80 with the I-
13 beam 142. Then the securing mechanism 42 can be transversally located along the I-
14 beam 142 and the alignment guides 250 are contacting the interior side of the trailer 20
15 contour frame 254. The predetermined distance 258 between the alignment guides 250
16 and the skirt-contacting portion 118, in consideration of the thickness of the contour
17 frame 254. The predetermined distance 258 can be calculated as follows: $T-t=d$ (T =
18 thickness of the contour frame 254; t =thickness of skirt panel 38; d = predetermined
19 distance 258). In some cases, the thickness of the contour frame 254 equals the
20 thickness of skirt panel 38 and the alignment guides 250 will be aligned with the skirt-
21 contacting portion 118 as illustrated in Figure 29. In some other cases the thickness of
22 the contour frame 254 is larger than the thickness of skirt panel 38 and the alignment
23 guides 250 will be protrude beyond the skirt-contacting portion 118 as illustrated in
24 Figure 34. In some additional cases the thickness of the contour frame 254 is smaller
25 than the thickness of skirt panel 38 and the alignment guides 250 will be aligned with
26 the skirt-contacting portion 118. The predetermined distance 258 is generally between
27 about 7 mm and about 12 mm. The alignment guides 250 have, for instance, a height of
28 between about 6 mm to about 25 mm.

29 **[0083]** As best seen in Figure 26, a plurality of sets of holes 126 are provided
30 with a distance 262 variation to accommodate different height of I-beams 142. The I-
31 beam 142 height can be different depending of the trailer 20. Tall I-beams 142.1, as

best seen in Figure 27, can be used to increase the strength of the trailer 20 to maximize the cargo load. For example, the portion of the trailer 20 accommodating the slider rail 24 can require shorter I-beam 142.2 to clear the contour frame 254 of the trailer 20. The higher set of holes 126.1 is going to be used in cooperation with a high I-beam 142.1 while the lower set of holes 126.2 is going to be used in cooperation with a low I-beam 142.2 as depicted in Figure 26.

[0084] The stabilizer 110 that can be seen in many Figures is an extension of the trailer-contacting portion 114 to extend the contact surface with the I-beam 142 to improve the strength of the upper portion 92 of the strut portion 84. As best seen in Figure 29, the transversal force 266, and-or the axial force 268, applied to the skirt panel 38 is transmitted to as a moment 270, a normal reaction 274 and/or an axial reaction 278 to the upper portion 92 of the strut portion 84. The stabilizer 110 function as a transfer lever element of the loads acting on the strut member 90 into the I-beam 142 on which the strut portion 84 is installed. In addition, the stress transferred towards the openings 130 locations due to deformation and flexing of the strut portion 84 is reduced by the stabilizer 110. The radius 282 of sweep of the stabilizer 110, illustrated in Figure 30 and Figure 31, is following a generally constant radius as the radius of the strut member 90 to optimize its compatibility with the strut member 90. The geometry of the strut member 90 is going to be discussed in further details below. Still referring to Figure 30, the trailer-contacting portion 114 includes a plurality of reinforcement ribs 290 oriented in different directions, generally orthogonal to each other, to help minimize deformation of the strut portion 84. The deformation in the Y direction is generally due to moment created by the weight of the skirt panel 38 and the stress applied to the strut portion 84 when contacting a foreign object. Ribs 290 in the X direction strengthen the skirt-contacting portion 118 from sagging due to the weight of the skirt panel 38 and the stress applied to the strut portion 84 when contacting a foreign object. The ribs 290 are preferably extending only from one surface, the upper surface or the lower surface, to facilitate the manufacturing process of the part. However, the surface of the stabilizer 110 that is in contact with the I-beam 142 has no ribs 290 to optimize its capacity to sustain significant pressure by spreading it over a larger contact surface and to avoid ribs 290 plastic deformation.

1 **[0085]** It can be appreciated from Figure 17 and Figure 29 that the strut portion
2 84 is sized and designed to allow connector portion 80 access through an access
3 portion 300 to access the fasteners 138 during installation of the securing mechanism
4 42. One can appreciate from Figure 29 the angle θ between the plane 304 where the
5 skirt panel 38 is going to be secured to the strut portion 84 and the axis 308 of the strut
6 member 90. The angle θ is between about 10 degree and about 30 degree. The angle θ
7 is preferably between about 12 degree and about 18 degree. Most preferably, the angle
8 θ is between about 15 degree and about 16 degree depending on the type of use and
9 the type of trailer 20.

10 **[0086]** A skilled reader would have noticed the shape of the strut member 90 that
11 includes a concave portion 320 (when seen from a side of the trailer 20 toward the
12 trailer 20). The concave portion 320 of the strut portion 90 offers some advantages in
13 the manner the strength of the strut portion 90 is applied to the skirt assembly 30 when
14 an external force, that might be caused by contacting an external object, is applied to
15 the strut portion 84. In light of Figure 36 throughout Figure 39, it can be noted that the
16 side of the cavity of the concave portion 320 has an impact on the amount of force
17 required to buckle and bend the strut member 90. It can be appreciated the cavity of the
18 concave portion 320 toward the skirt panel 38 provides more strength when a force is
19 applied toward the trailer 20. In case another behavior requiring a smaller force to bend
20 the strut member 90 is desirable, the concave portion 320 could alternatively be
21 inverted toward the center of the trailer without departing from the scope of the present
22 description. In the present situation where the cavity of the concave portion 320 is
23 located toward the skirt panel 38, a stronger force directed toward the trailer is required
24 to bend the member 90 than if the cavity of the concave portion 320 is in the opposite
25 direction toward the center of the trailer 20. The concave portion 320 is going to change
26 progressively as illustrated in Figure 37 and the section of the concave portion 320
27 tends to straighten 346, as illustrated by arrows 324. The resistance of the strut member
28 90 will be strong until the strut member 90 buckles 330 and flexes more easily after it
29 has buckled as illustrated in Figure 39. This offers significant advantages by maintaining
30 strongly the skirt panel 38 in an aerodynamically operating configuration 334 that could

flex and provide some deflection 338 until a predetermined load is applied that overcome the buckling load threshold 330 of the strut member 90 design. In other words, the strut member 90 is going to be strong and maintain an efficient operating configuration despite the loads applied thereon when the loads are smaller than the buckling load threshold 330 and will become less resistant when buckling and more prone to flex and deflect significantly 342 when the load applied thereon is beyond the buckling load threshold 330. Such a significant load is generally caused by an undesirable event, like a contact with an exterior object, to overcome the object without damaging the aerodynamic skirt assembly 30. The operating configuration 334 position of the strut member 90 is automatically recovered when the load is significantly reduced, or removed. Table A below is in reference with Figure 37.

Strut member's	Minimum	Maximum
Depth A	5 mm	25 mm
Depth B	6 mm	20 mm
Depth C	7 mm	15 mm
Depth D	8 mm	12 mm
Width A	50 mm	180 mm
Width B	60 mm	160 mm
Width C	70 mm	140 mm
Width D	80 mm	120 mm
Thickness A	2 mm	10 mm
Thickness B	3 mm	8 mm
Thickness C	4 mm	7 mm
Thickness D	4.5 mm	6 mm

Table A

[0087] To achieve that, the strut portion 84 can be built from a variety of polymers that are flexible and resilient enough to perform the required flex. Materials such as, but not limited to, polypropylene, nylon, nylon co-polymer, thermoplastic poly olefin, polyethylene, polycarbonate and thermosets, charged with fibers or not, could be used.

1 Alternatively, other material such as metal that are designed to remain in their elastic
2 domain can also be suitable to bend, buckle and automatically recover its original
3 shape. Manufacturing processes including, but not limited to, injection molding,
4 machining, thermoforming and RTM could be used in consideration of the selected
5 material and other constraints.

6 **[0088]** An additional embodiment is illustrated in Figure 40 and Figure 41. In this
7 embodiment, the strut member 90 includes two concave portions 320.1, 320.2 directed
8 toward opposite directions. A transition portion 350 where the two concave portions
9 320.1, 320.2 are merging is illustratively located around the middle portion of the strut
10 member 90. The transition portion 350 delimits a portion of the strut member 90 where
11 the flex behavior of the strut member 90 is not within the desired load-supporting range
12 of either of the two concave portions 320.1, 320.2. For instance, the transition portion
13 350 includes the portion of the strut member 90 that is not concave and offers much
14 less resistance to buckling and bending. The size of the transition portion 350 can vary
15 depending of the particular design of the strut member 90. An advantage of the
16 embodiment illustrated in Figure 40 and Figure 41 would be to provide a comparable
17 resistance when the skirt panel 38 sustains a force in the direction of the trailer 20 and a
18 force in the direction opposed to the trailer 20, taking advantages of the location and the
19 direction of the cavities 320.1, 320.2 to allow resiliency behaviors in both directions.
20 Figure 40 illustrates a rather flat transition portion 350 while Figure 41 illustrates an
21 alternate design including a transition portion with an offset portion 354 to ensure one
22 edge of each of the two concave portions 320.1, 320.2 are co-linear to help
23 standardizing the behavior of both sides of the strut member 90.

24 **[0089]** The transition portion 350 can alternatively include reinforcing ribs to
25 prevent bending. Ribs (not illustrated) can be added on each edges of the strut member
26 90 over the length of the transition portion 350. Additional ribs can be localized over the
27 surface of the transition portion 350. Alternatively, a pair of clamp members (not
28 illustrated) can be added to sandwich the transition portion 350 and secured together
29 with fasteners, for example. The pair of clamp members prevents bending in the region
30 of the strut member 90 that is covered by the pair of clamp members. The design,

1 shape, length and location of the pair of clamp members can be used to modify, adjust
2 and optimize the behavior of the strut portion 84.

3 **[0090]** While the invention has been described in connection with what is
4 presently considered to be the most practical and preferred embodiments, it is to be
5 understood that the invention is not to be limited to the disclosed embodiments and
6 elements, but, to the contrary, is intended to cover various modifications, combinations
7 of features, equivalent arrangements, and equivalent elements included within the spirit
8 and scope of the appended claims. Furthermore, the dimensions of features of various
9 components that may appear on the drawings are not meant to be limiting, and the size
10 of the components therein can vary from the size that may be portrayed in the figures
11 herein. Thus, it is intended that the present invention covers the modifications and
12 variations of the invention, provided they come within the scope of the appended claims
13 and their equivalents.

14

What is claimed is:

1. A skirt securing connector assembly for securing an aerodynamic skirt panel to a trailer, the skirt securing connector assembly comprising:
 - an upper attachment portion adapted to secure the skirt securing connector assembly to the trailer;
 - a lower attachment portion adapted to secure a lower portion of the aerodynamic skirt panel; and
 - a strut member connected, at a first end thereof, to the upper portion of the skirt securing connector assembly and connected, at a second end thereof, to the lower portion of the skirt securing connector assembly, the strut member including a longitudinal axis between the first end and the second end, the strut member including a concave portion extending along the longitudinal axis, the concave portion increasing an inertia moment of the strut member to sustain a force applied by air routing on the aerodynamic skirt in an aerodynamic configuration and allowing the strut member to deform to buckle in a buckled configuration and bend when a force exceeding a buckling threshold of the strut member is applied on the aerodynamic skirt to momentarily move the aerodynamic skirt from the aerodynamic configuration to an object avoidance configuration and moving back by itself to the aerodynamic configuration when the force exceeding the buckling threshold of the strut member is not applied on the aerodynamic skirt.
2. The skirt securing connector assembly of claim 1, wherein the strut member is adapted to buckle on an upper half of the strut member.
3. The skirt securing connector assembly of claim 1, wherein the strut member, in the buckled configuration, includes an angle of less than 180-degree between an upper portion of the strut member and a lower portion of the strut member, in a direction opposed to the aerodynamic skirt panel.
4. The skirt securing connector assembly of claim 1, wherein the concave portion includes a cavity substantially facing the aerodynamic skirt.

5. The skirt securing connector assembly of claim 1, wherein the upper attachment portion is adapted to secure the skirt securing connector assembly to an upper portion of the aerodynamic skirt panel.

6. The skirt securing connector assembly of claim 5, wherein the upper attachment portion of the skirt securing connector assembly includes a planar portion for securing thereon the aerodynamic skirt panel and wherein the lower attachment portion includes a planar portion for securing thereon the aerodynamic skirt panel, both planar portions being parallel in respect to one another.

7. The skirt securing connector assembly of claim 1, wherein the strut member includes a first elasticity modulus in a non-buckled configuration and a second elasticity modulus in the buckled configuration.

8. The skirt securing connector assembly of claim 7, wherein the strut member, in the non-buckled configuration, is sustaining a load that is more than twice the load sustained by the strut member in the non-buckled configuration.

9. The skirt securing connector assembly of claim 1, wherein a cross-section shape of the strut member changes from a first cross-section shape in a non-buckled configuration to a second cross-section shape in the buckled configuration.

10. The skirt securing connector assembly of claim 1, wherein the skirt securing connector assembly includes a pair of skirt-securing portions laterally disposed in respect with the strut member longitudinal axis.

11. An aerodynamic skirt assembly comprising:
an aerodynamic skirt panel;
a plurality of skirt securing connectors assemblies, each skirt securing connector assembly comprising
an upper attachment portion adapted to secure the skirt securing connector assembly to the trailer;
a lower attachment portion adapted to secure a lower portion of the aerodynamic skirt panel; and

1 a strut member connected, at a first end thereof, to the upper portion of the skirt
2 securing connector assembly and connected, at a second end thereof, to the lower
3 portion of the skirt securing connector assembly, the strut member including a
4 longitudinal axis between the first end and the second end, the strut member
5 including a concave portion extending along the longitudinal axis, the concave
6 portion increasing an inertia moment of the strut member to sustain a force applied
7 by air routing on the aerodynamic skirt in an aerodynamic configuration and
8 allowing the strut member to deform to buckle in a buckled configuration and bend
9 when a force exceeding a buckling threshold of the strut member is applied on the
10 aerodynamic skirt to momentarily move the aerodynamic skirt from the
11 aerodynamic configuration to an object avoidance configuration and moving back
12 by itself to the aerodynamic configuration when the force exceeding the buckling
13 threshold of the strut member is not applied on the aerodynamic skirt.

14 12. The aerodynamic skirt assembly of claim 11, wherein the strut member is
15 adapted to buckle on an upper half of the strut member.

16 13. The aerodynamic skirt assembly of claim 11, wherein the strut member, in the
17 buckled configuration, includes an angle of less than 180-degree between an upper
18 portion of the strut member and a lower portion of the strut member, in a direction
19 opposed to the aerodynamic skirt panel.

20 14. The aerodynamic skirt assembly of claim 11, wherein the concave portion
21 includes a cavity substantially facing the aerodynamic skirt.

22 15. The aerodynamic skirt assembly of claim 11, wherein the upper attachment
23 portion is adapted to secure the skirt securing connector assembly to an upper portion
24 of the aerodynamic skirt panel.

25 16. The aerodynamic skirt assembly of claim 15, wherein the upper attachment
26 portion of the skirt securing connector assembly includes a planar portion for securing
27 thereon the aerodynamic skirt panel and wherein the lower attachment portion includes

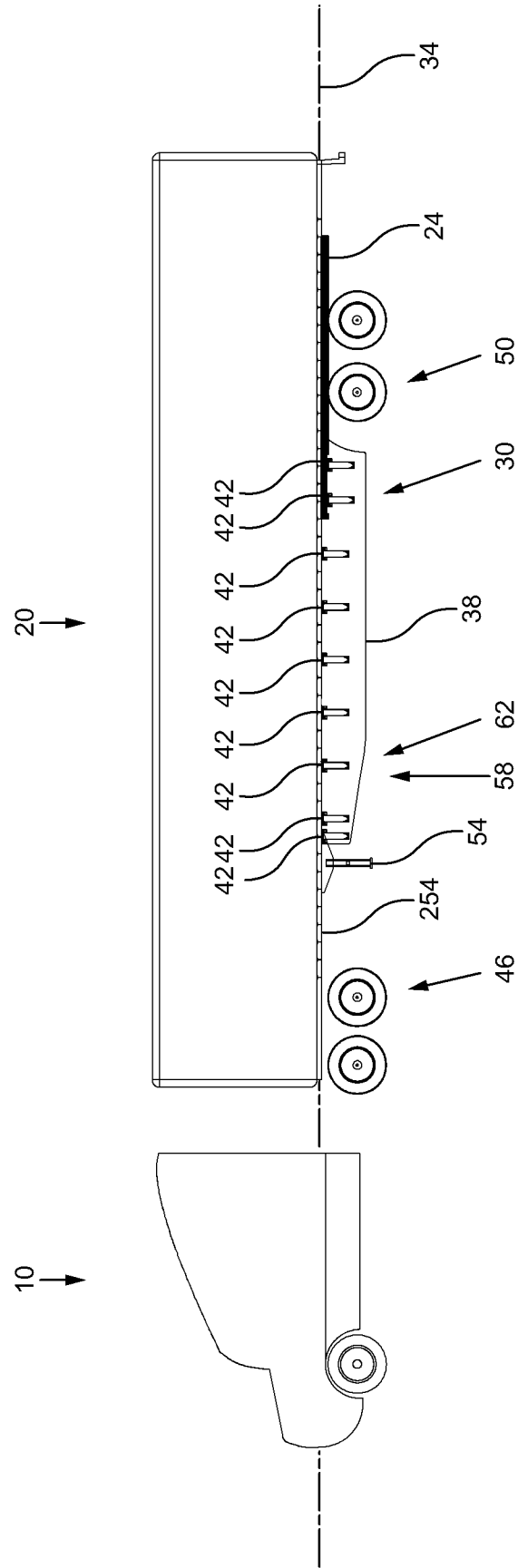
1 a planar portion for securing thereon the aerodynamic skirt panel, both planar portions
2 being parallel in respect to one another.

3 17. The aerodynamic skirt assembly of claim 11, wherein the strut member includes
4 a first elasticity modulus in a non-buckled configuration and a second elasticity modulus
5 in a buckled configuration.

6 18. The aerodynamic skirt assembly of claim 17, wherein the strut member, in the
7 non-buckled configuration, is sustaining a load that is more than twice the load
8 sustained by the strut member in the non-buckled configuration.

9 19. The aerodynamic skirt assembly of claim 11, wherein a cross-section shape of
10 the strut member changes from a first cross-section shape in a non-buckled
11 configuration to a second cross-section shape in the buckled configuration.

12 20. The aerodynamic skirt assembly of claim 11, wherein the skirt securing
13 connector assembly includes a pair of skirt-securing portions laterally disposed in
14 respect with the strut member longitudinal axis.

**FIG. 1**

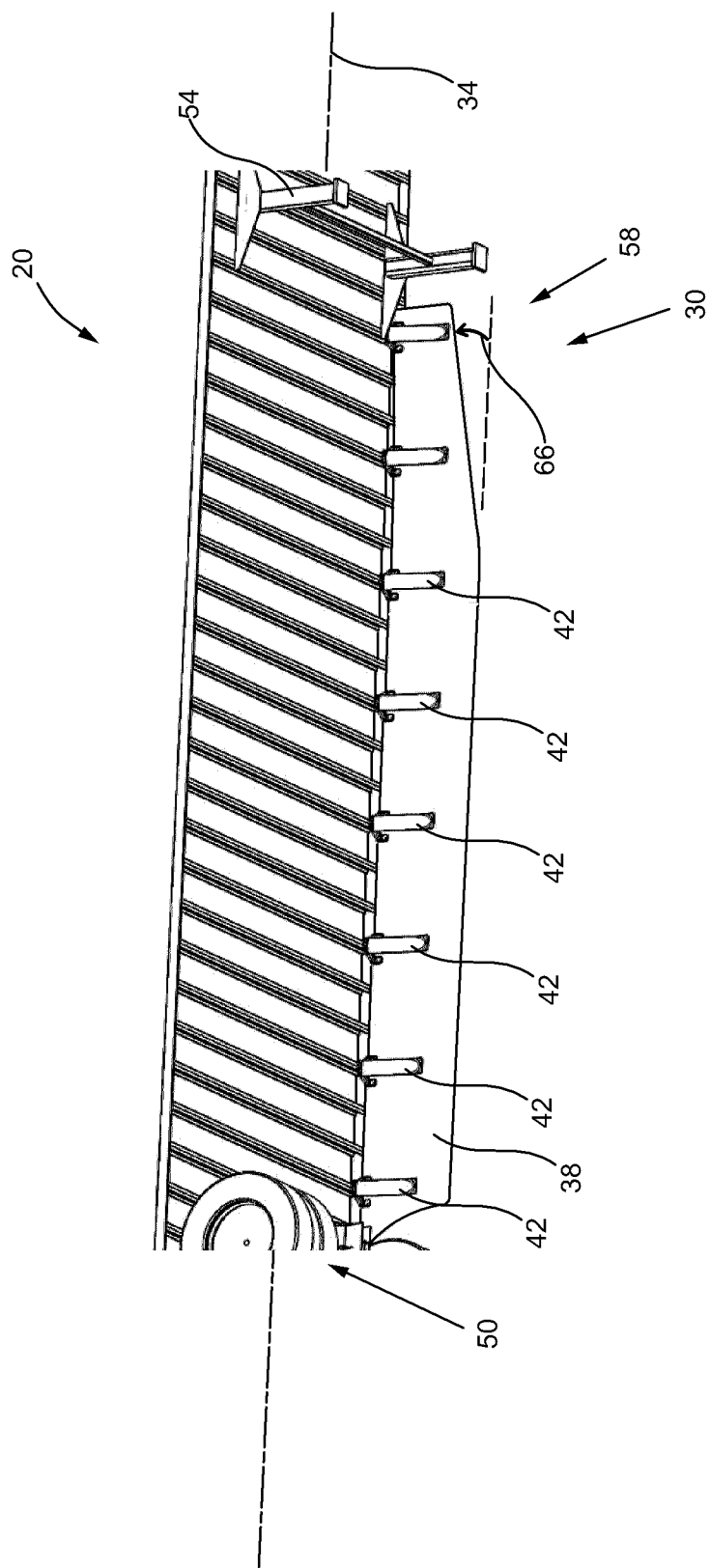


FIG. 2

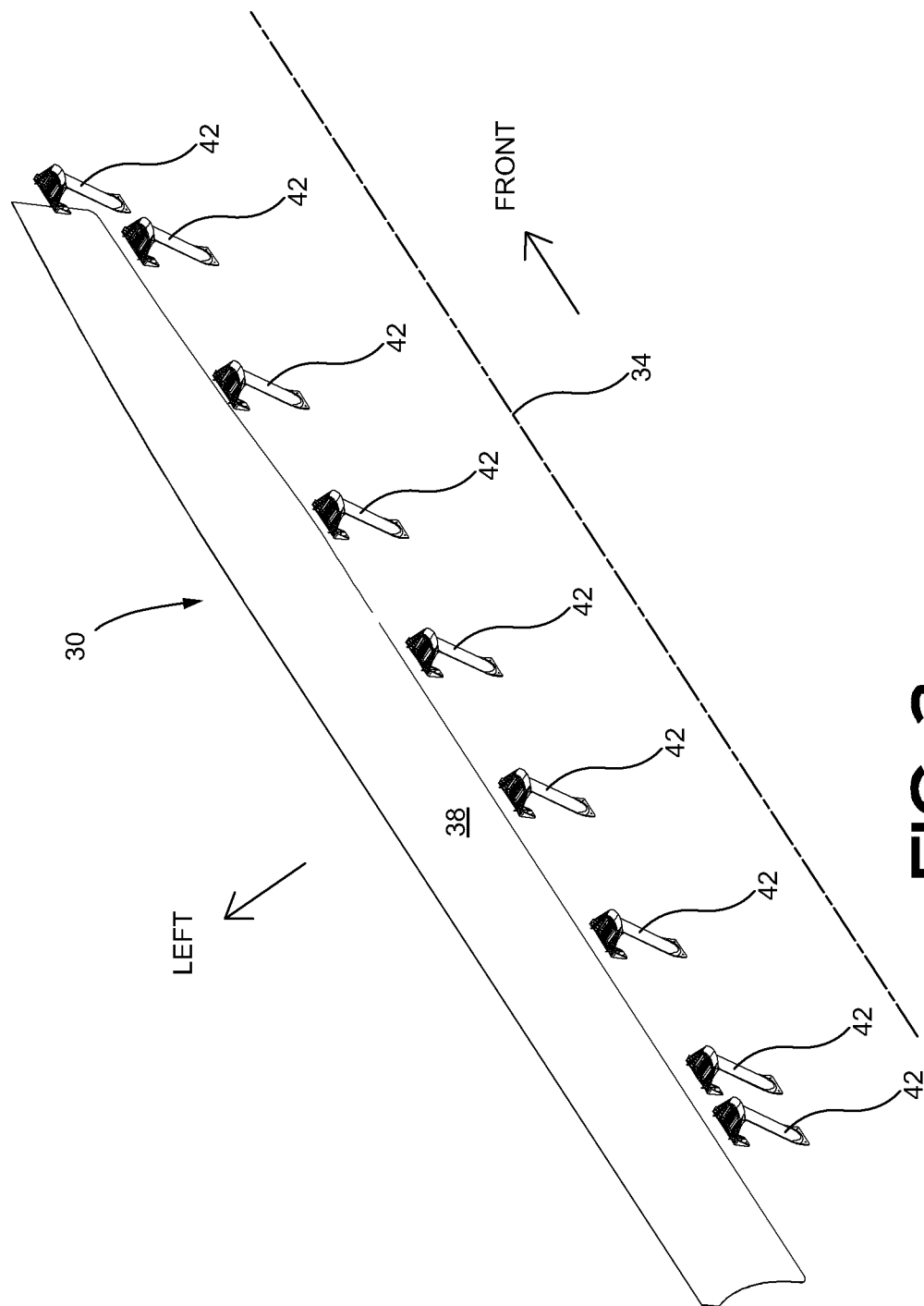


FIG. 3

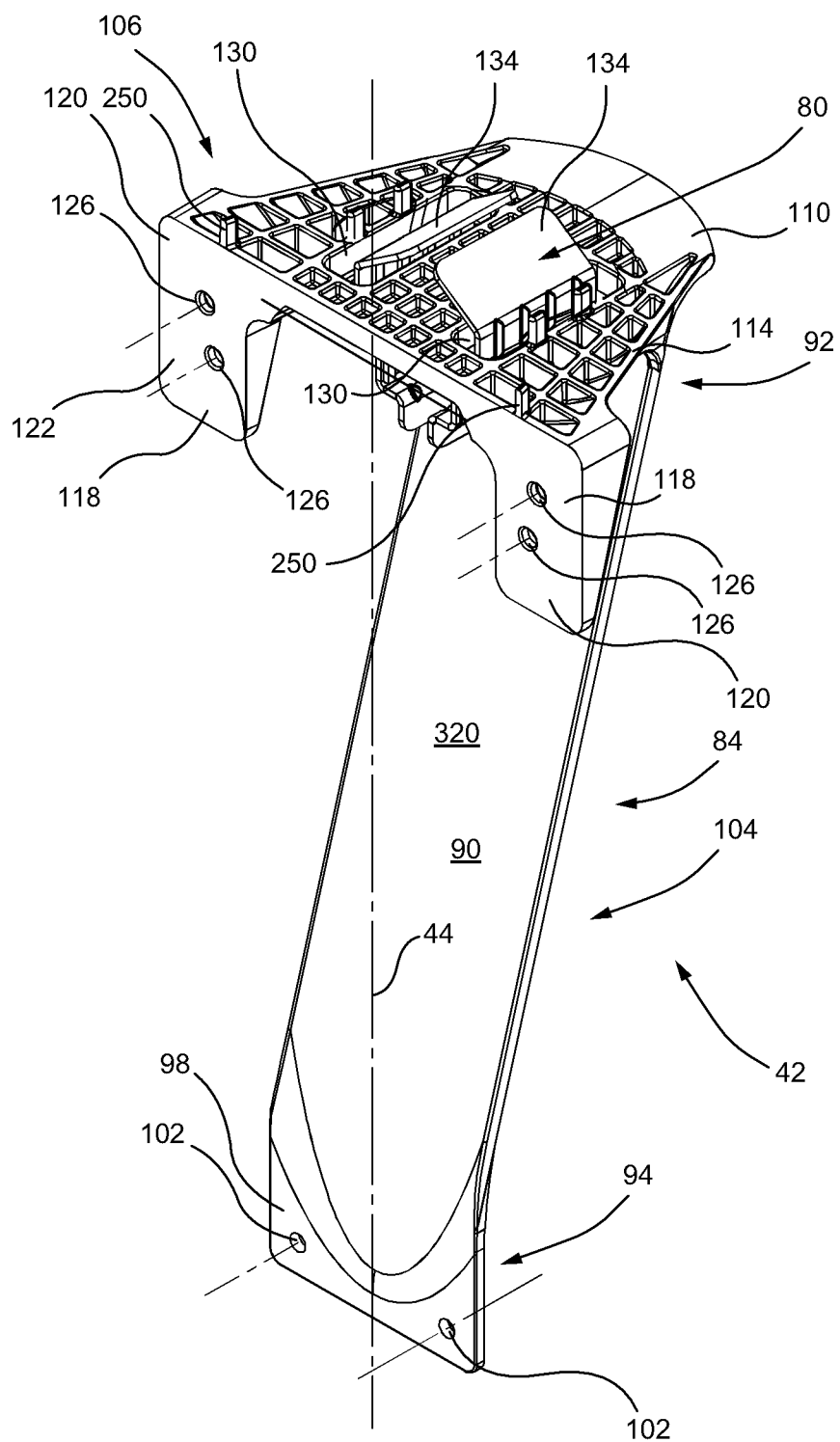


FIG. 4

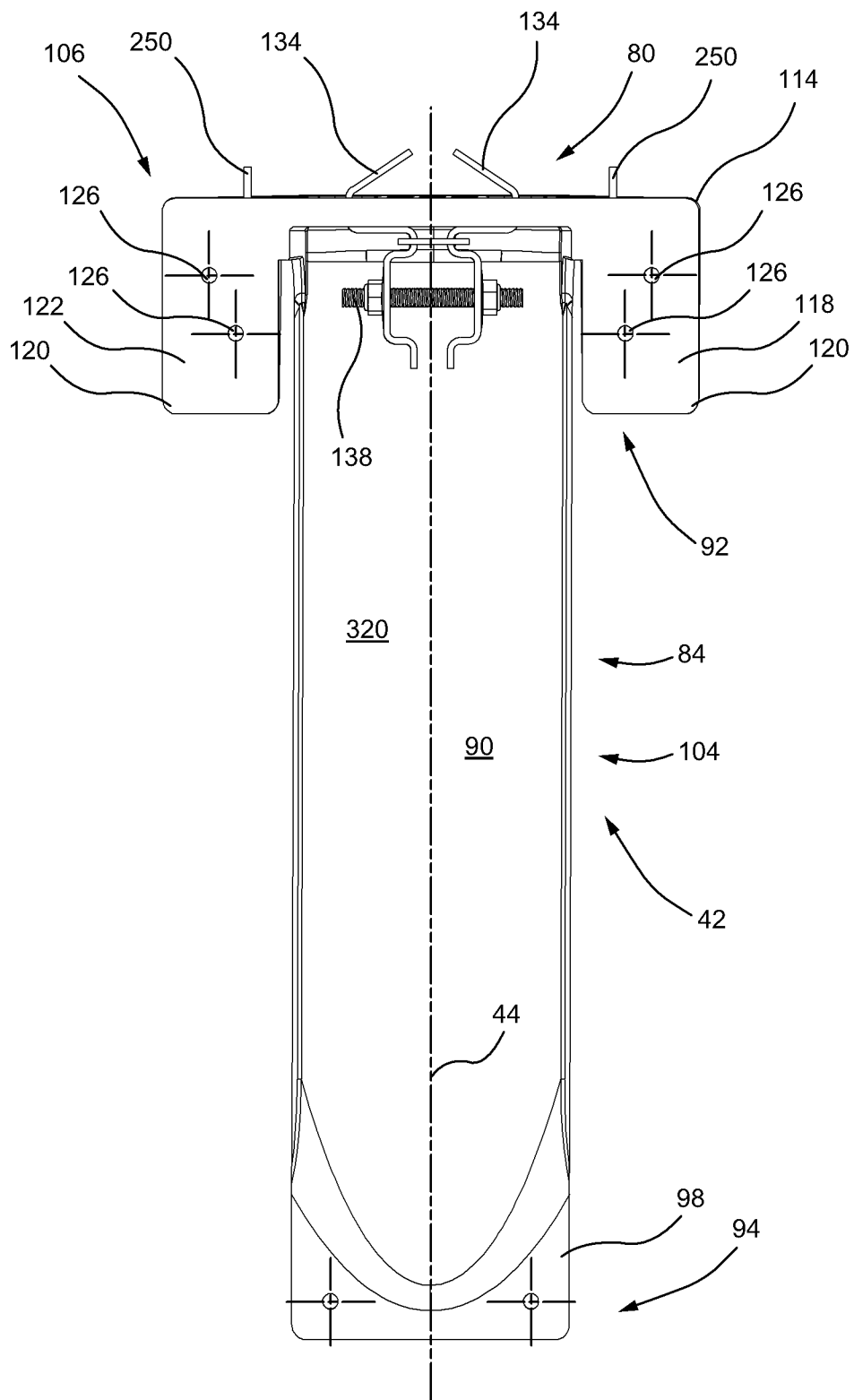


FIG. 5

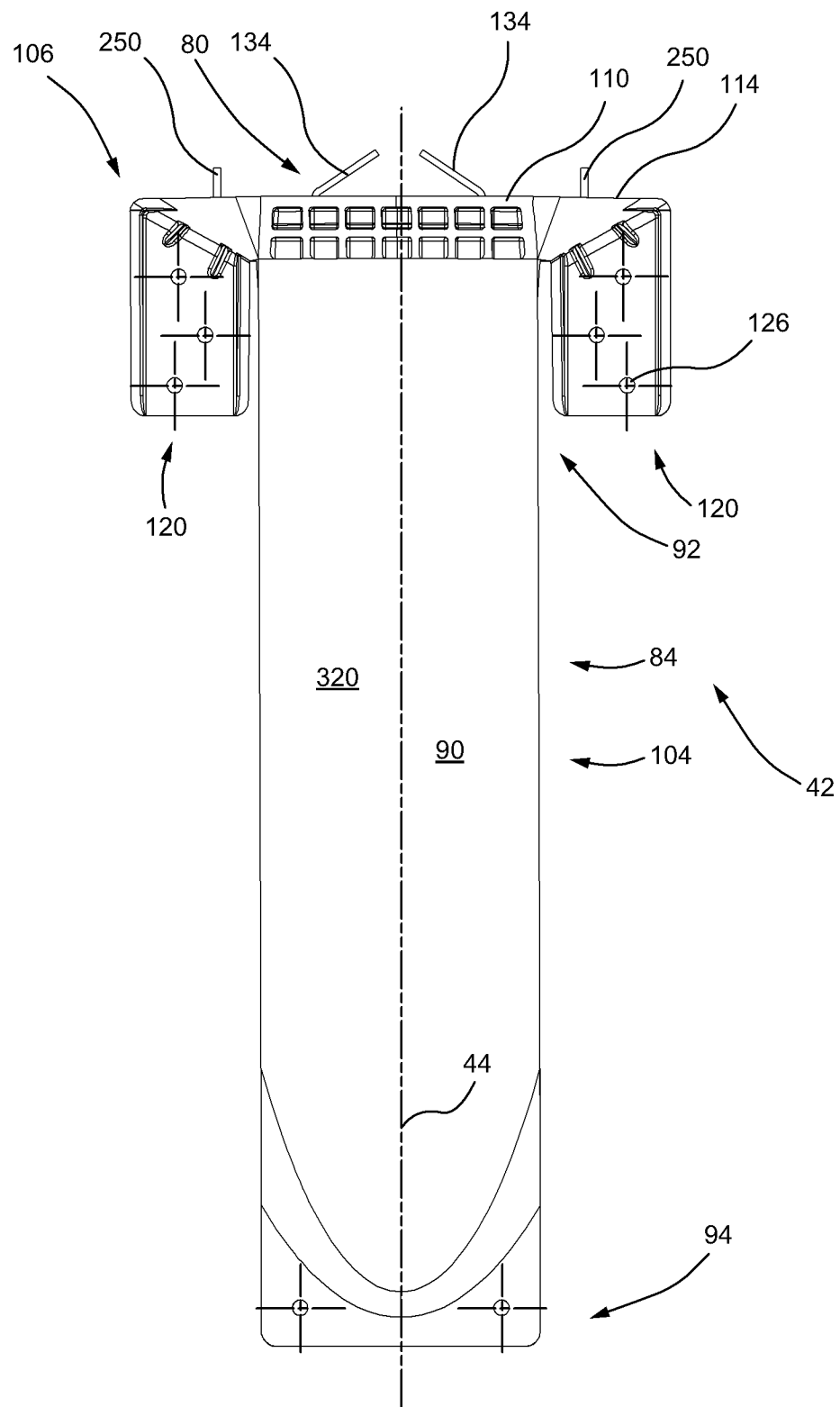


FIG. 6

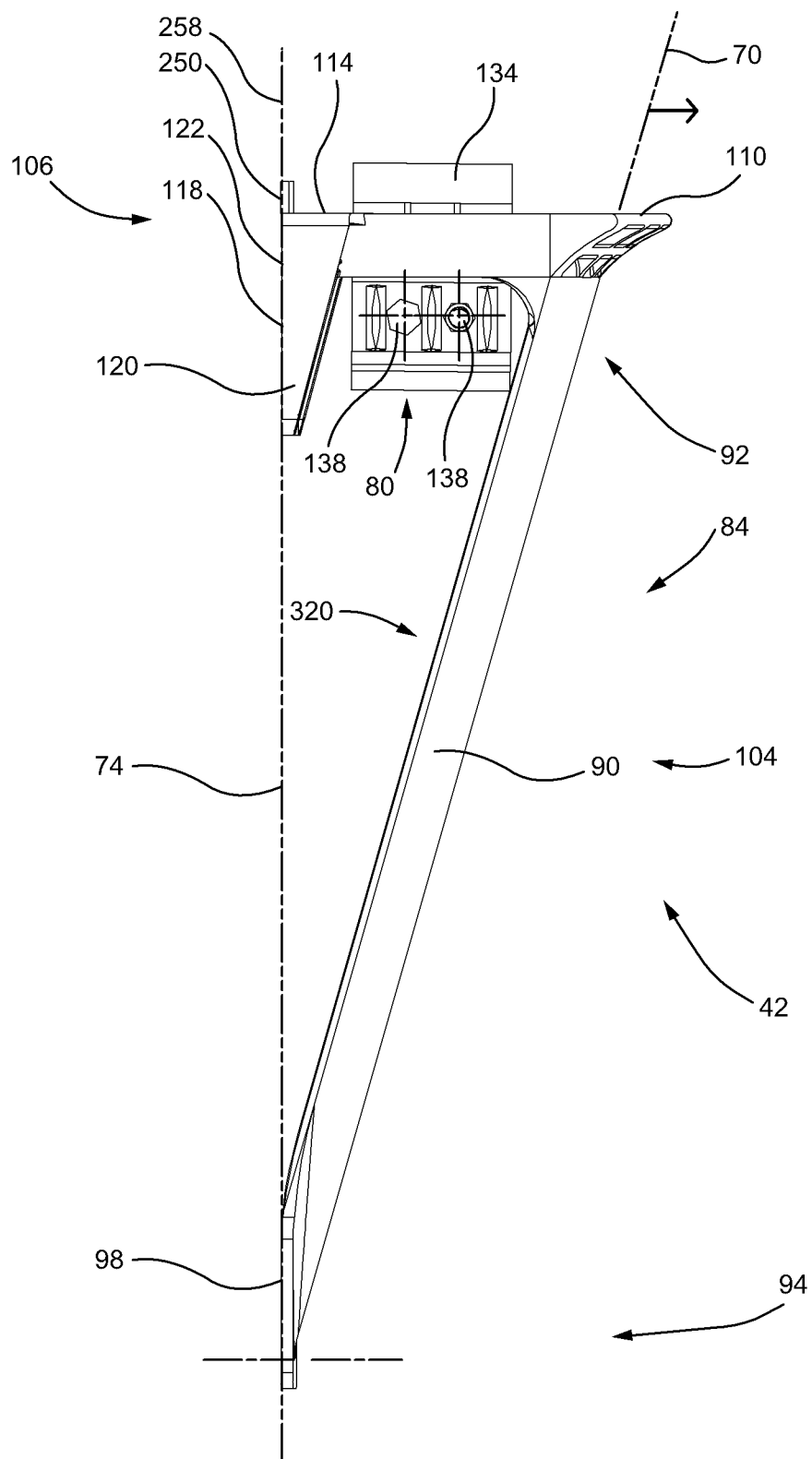


FIG. 7

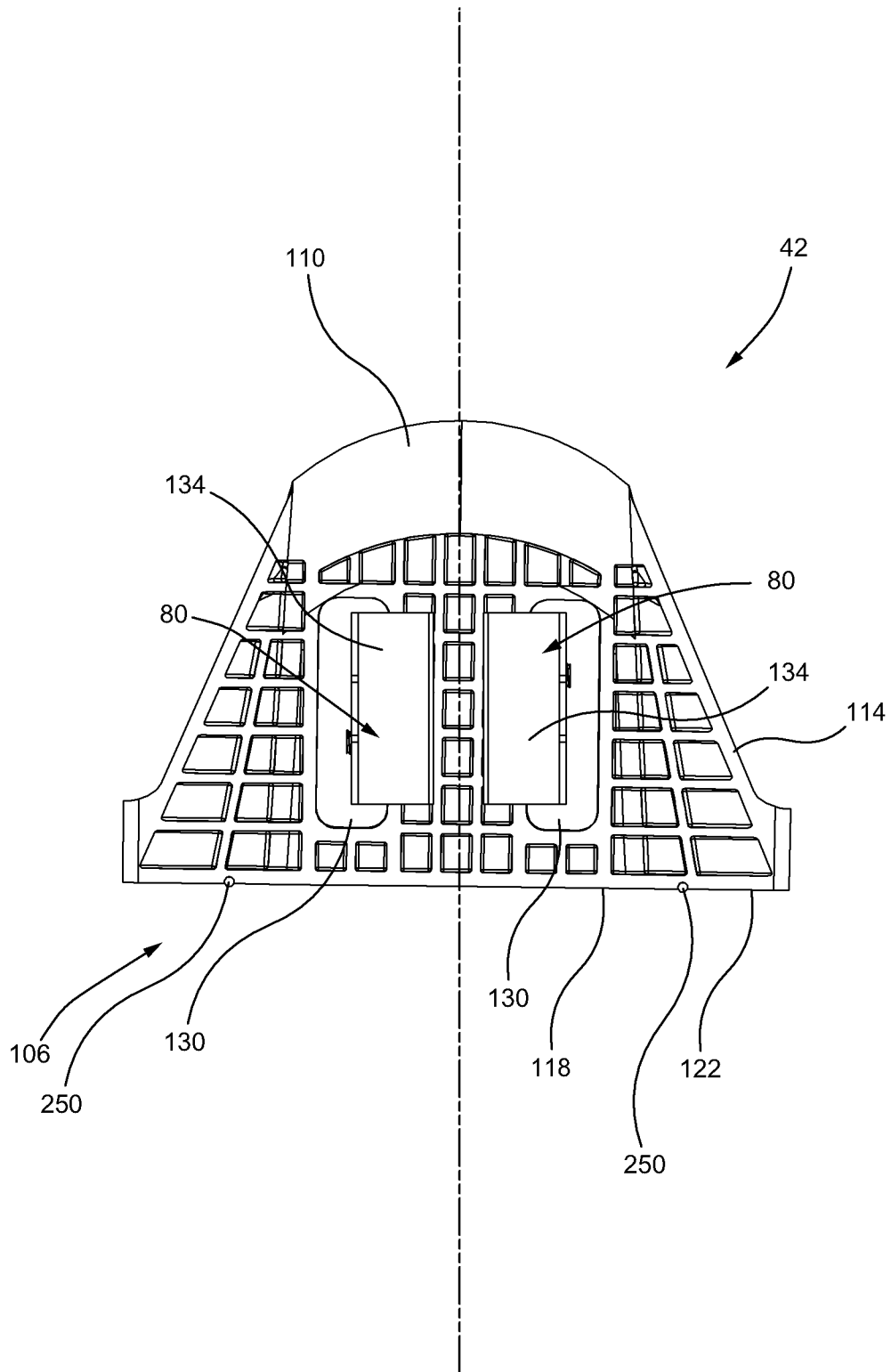


FIG. 8



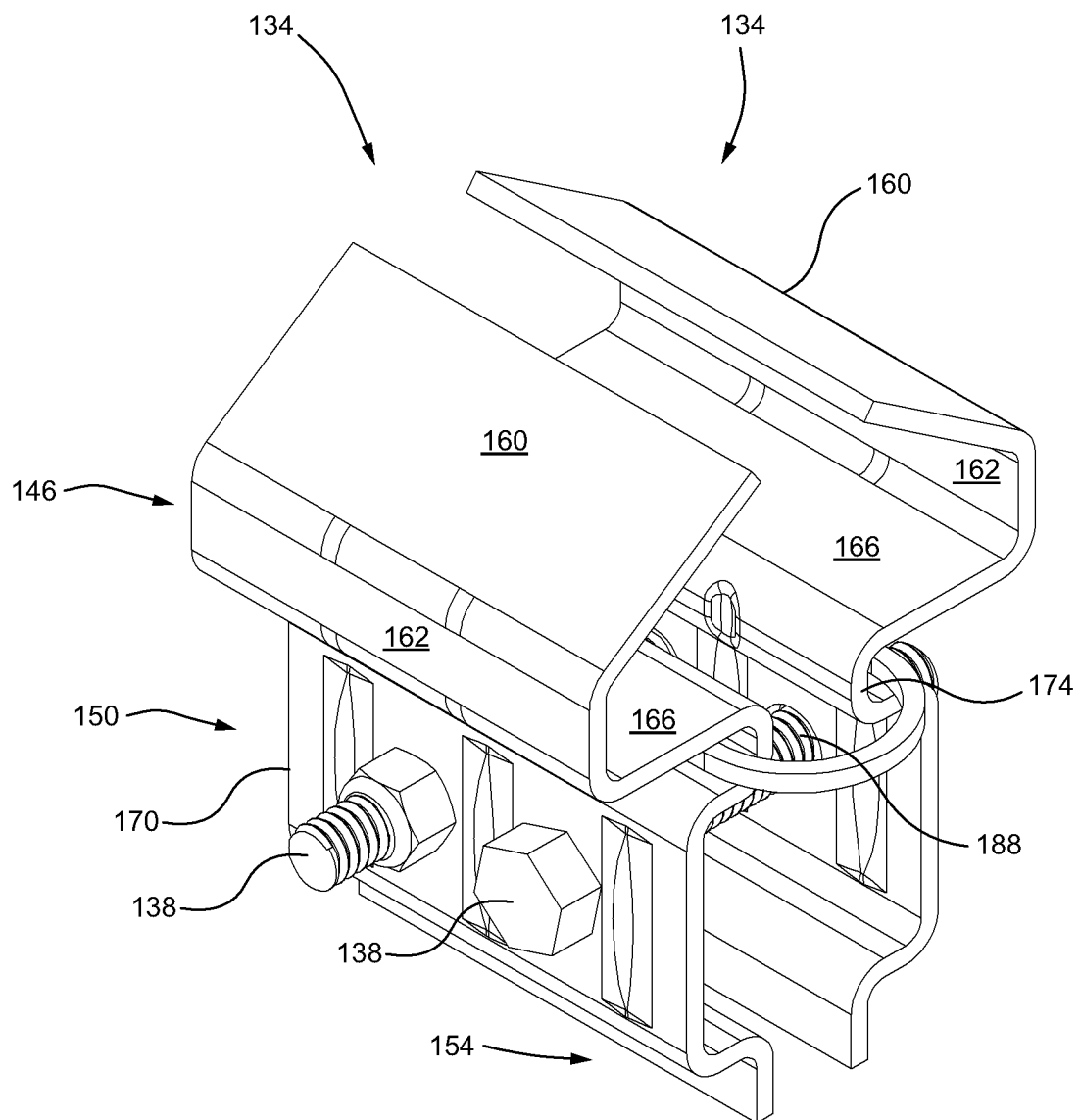


FIG. 10

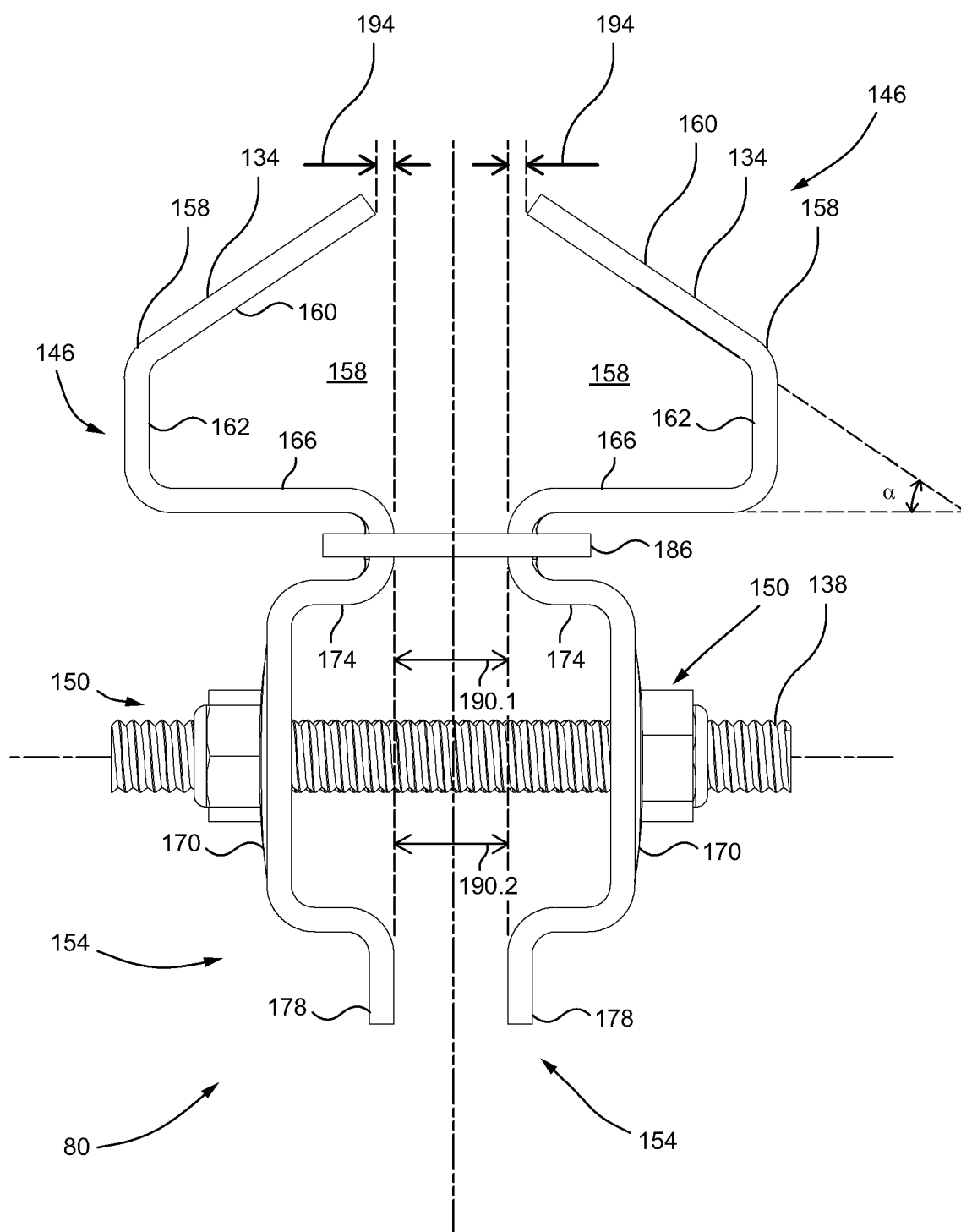


FIG. 11

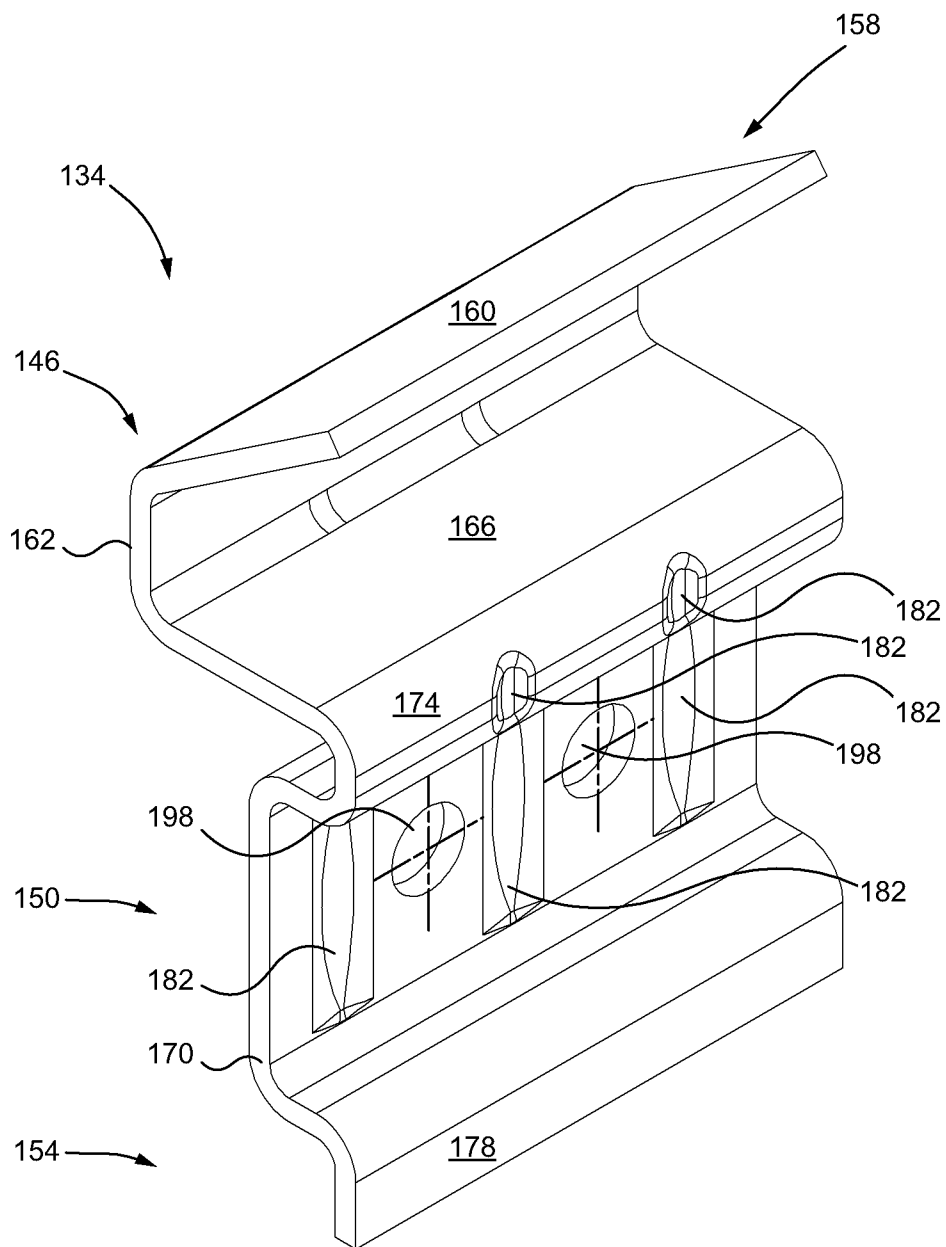


FIG. 12

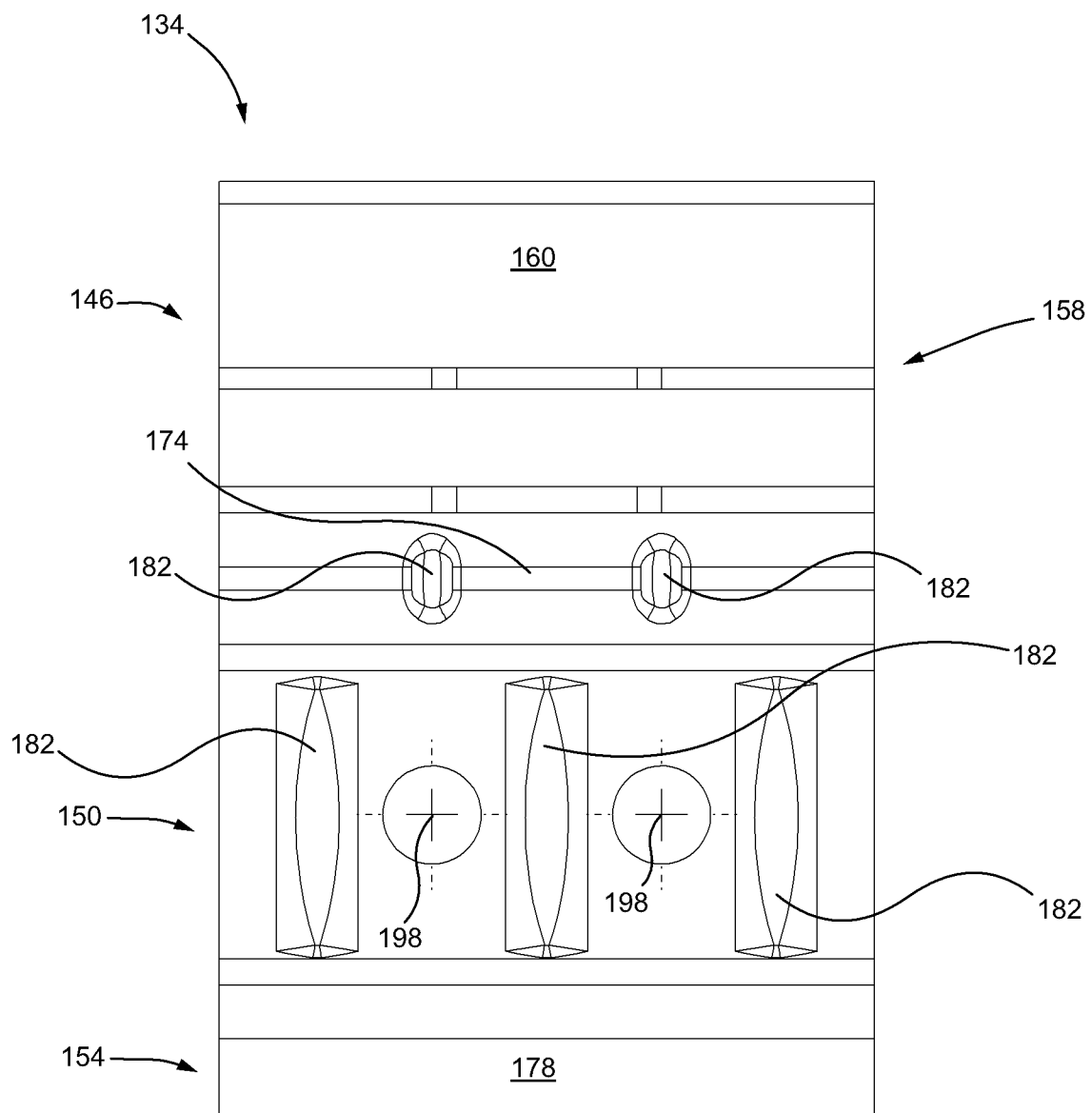


FIG. 13

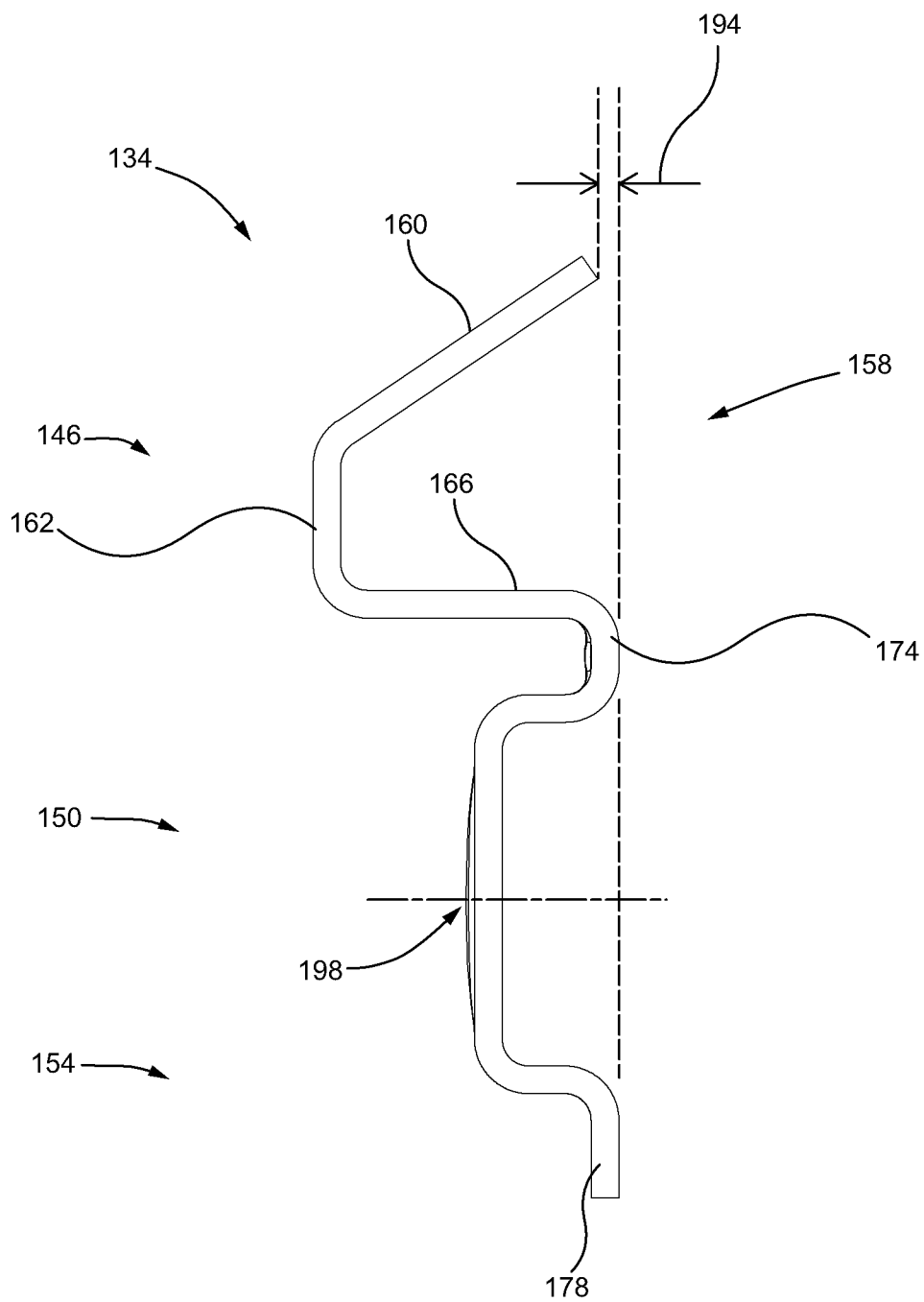


FIG. 14

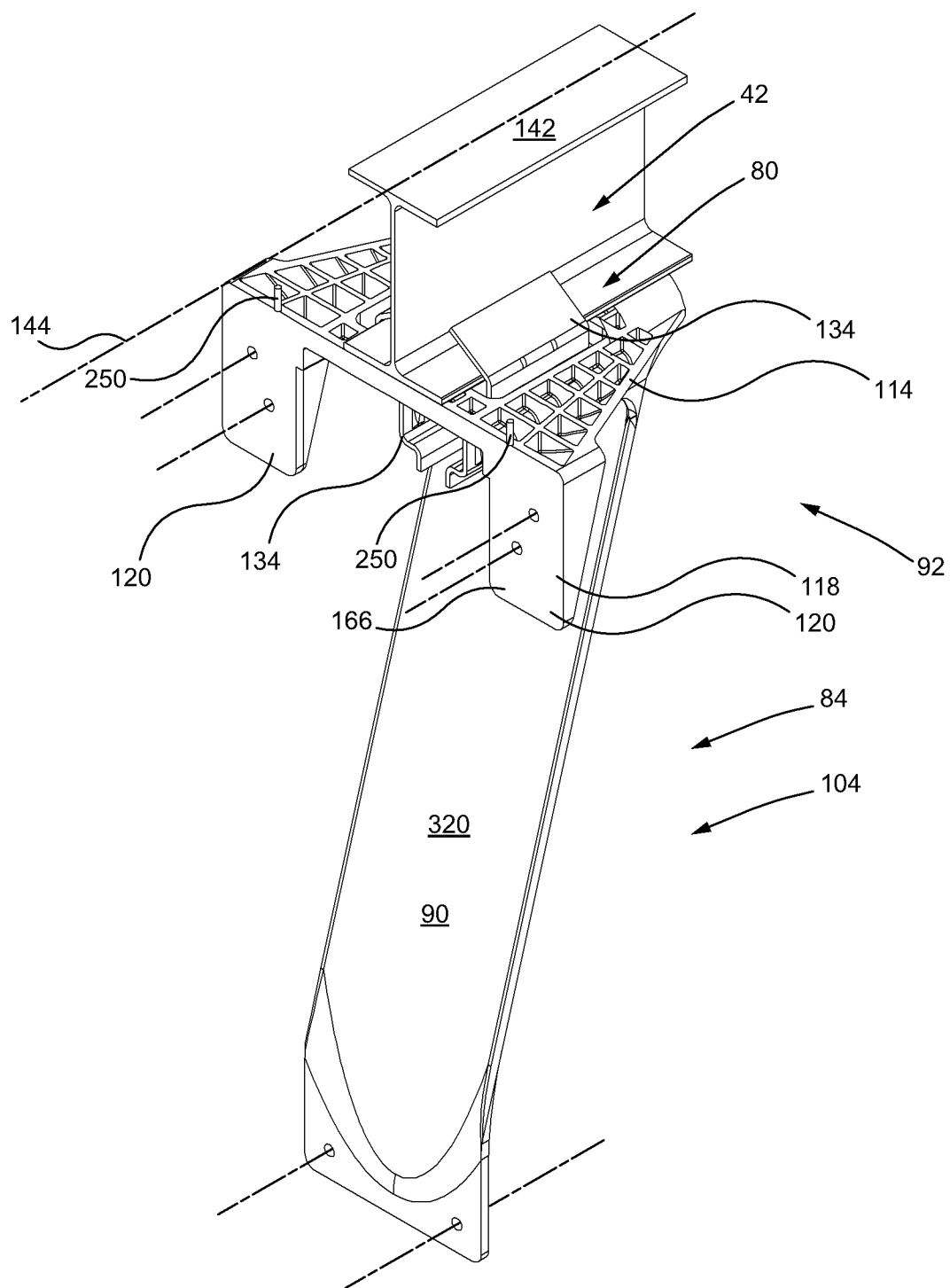


FIG. 15





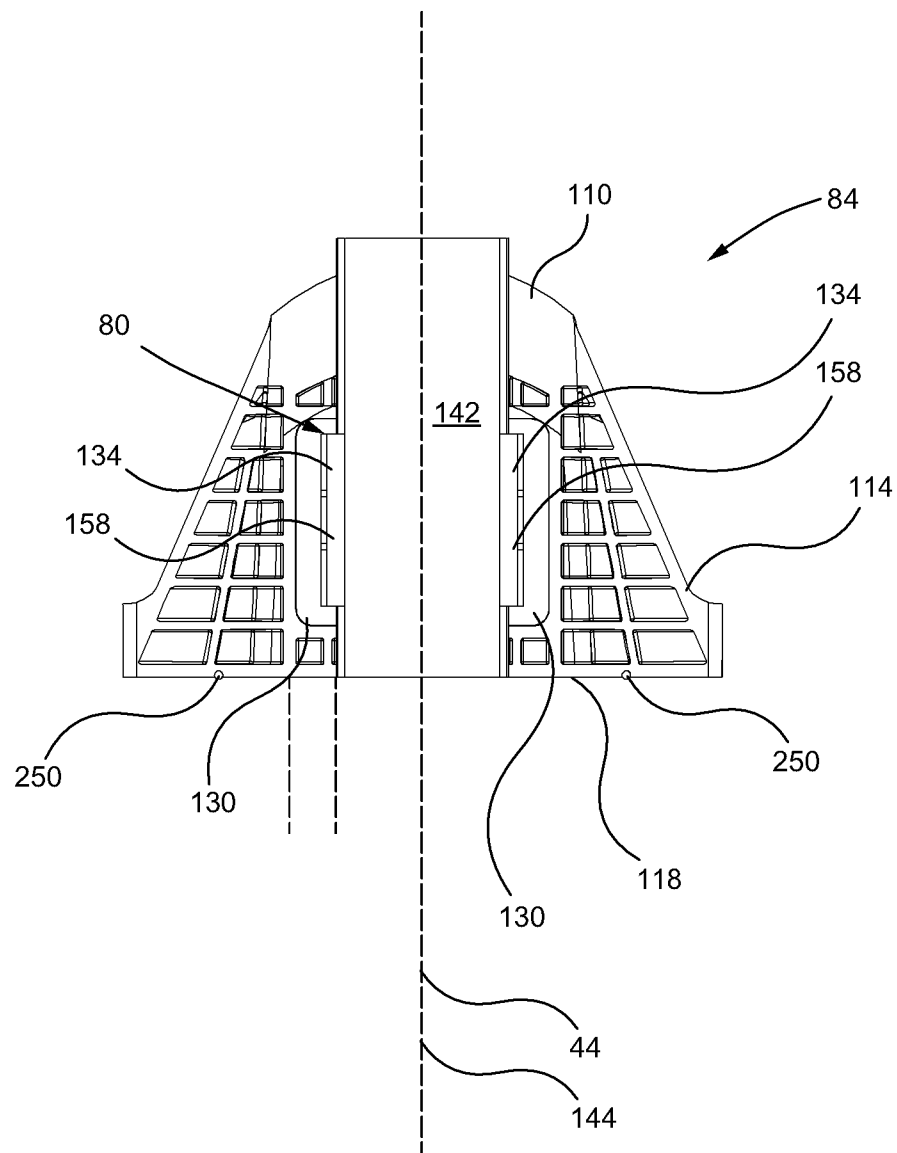


FIG. 18

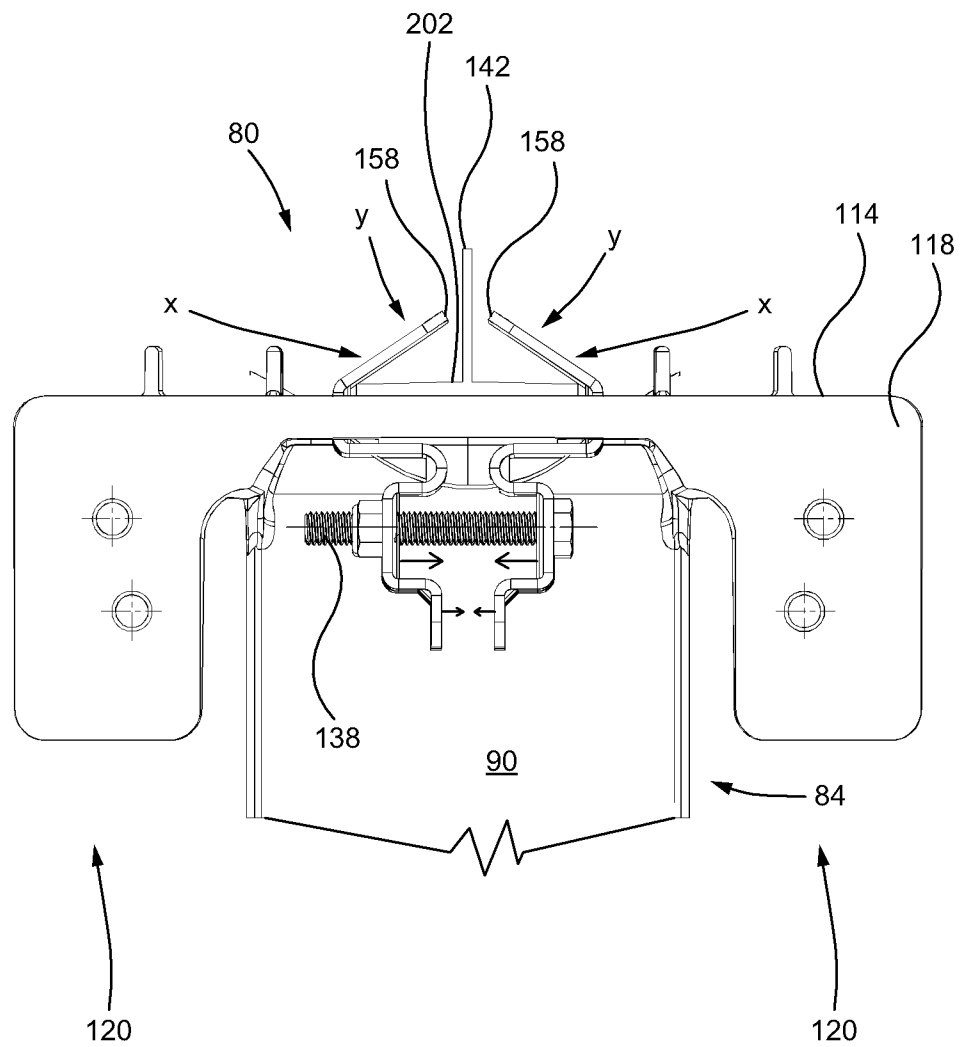


FIG. 19

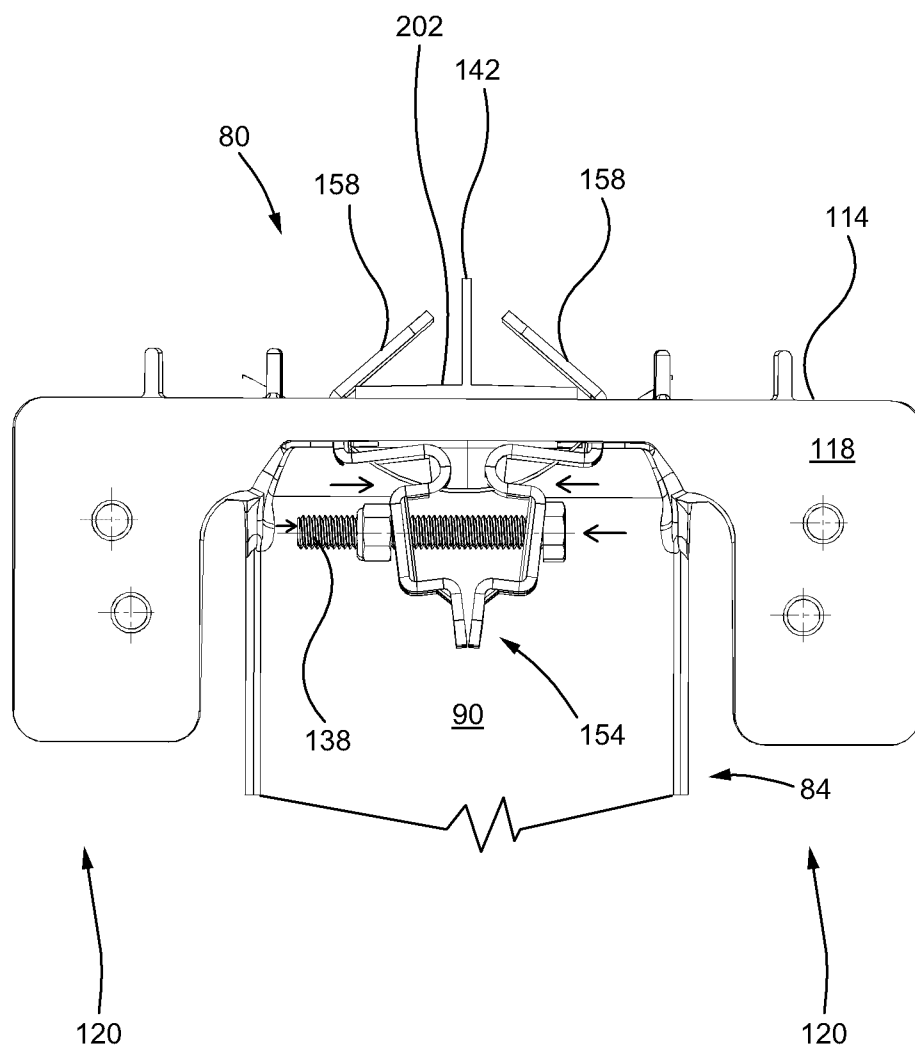


FIG. 20

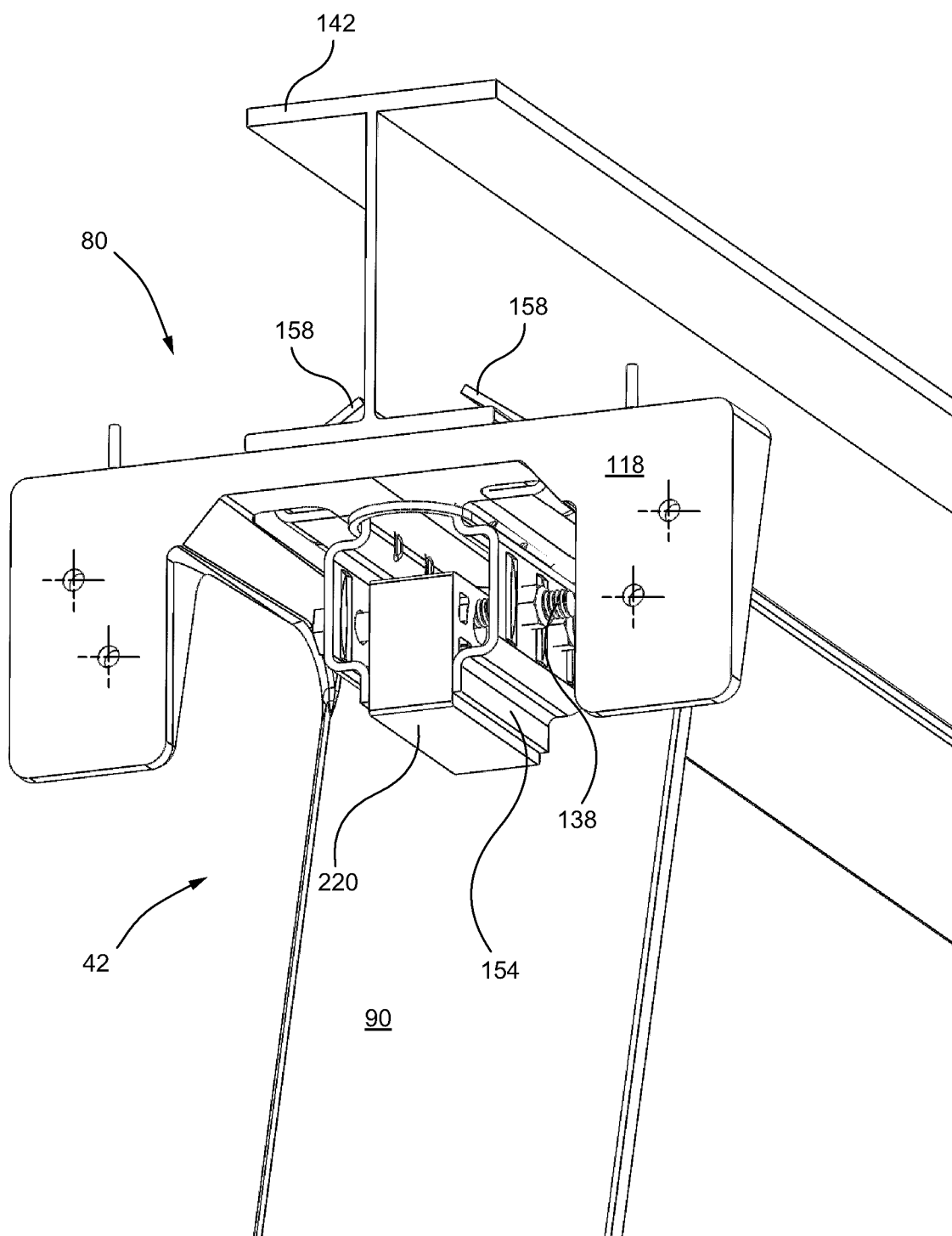


FIG. 21

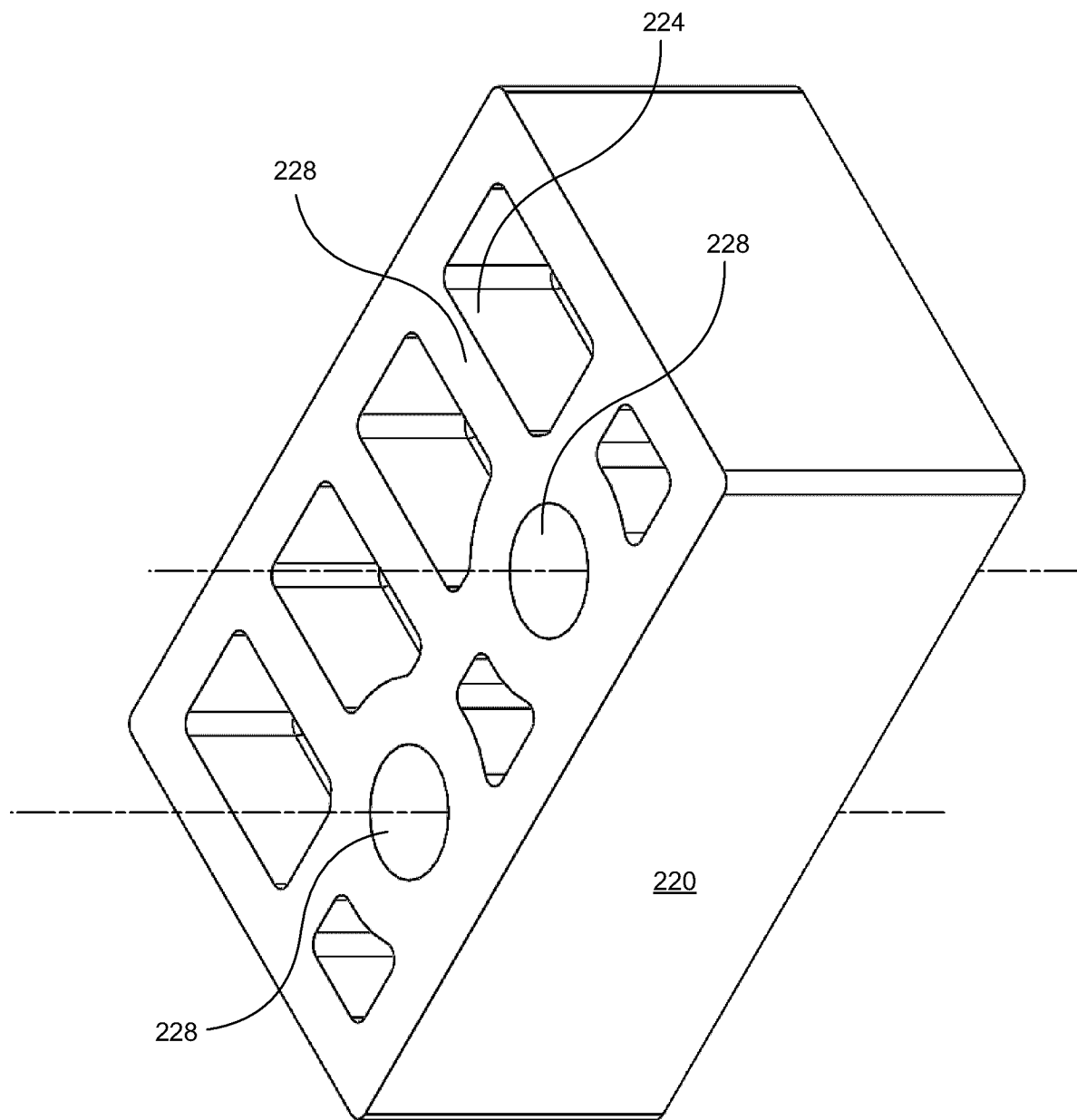


FIG. 22

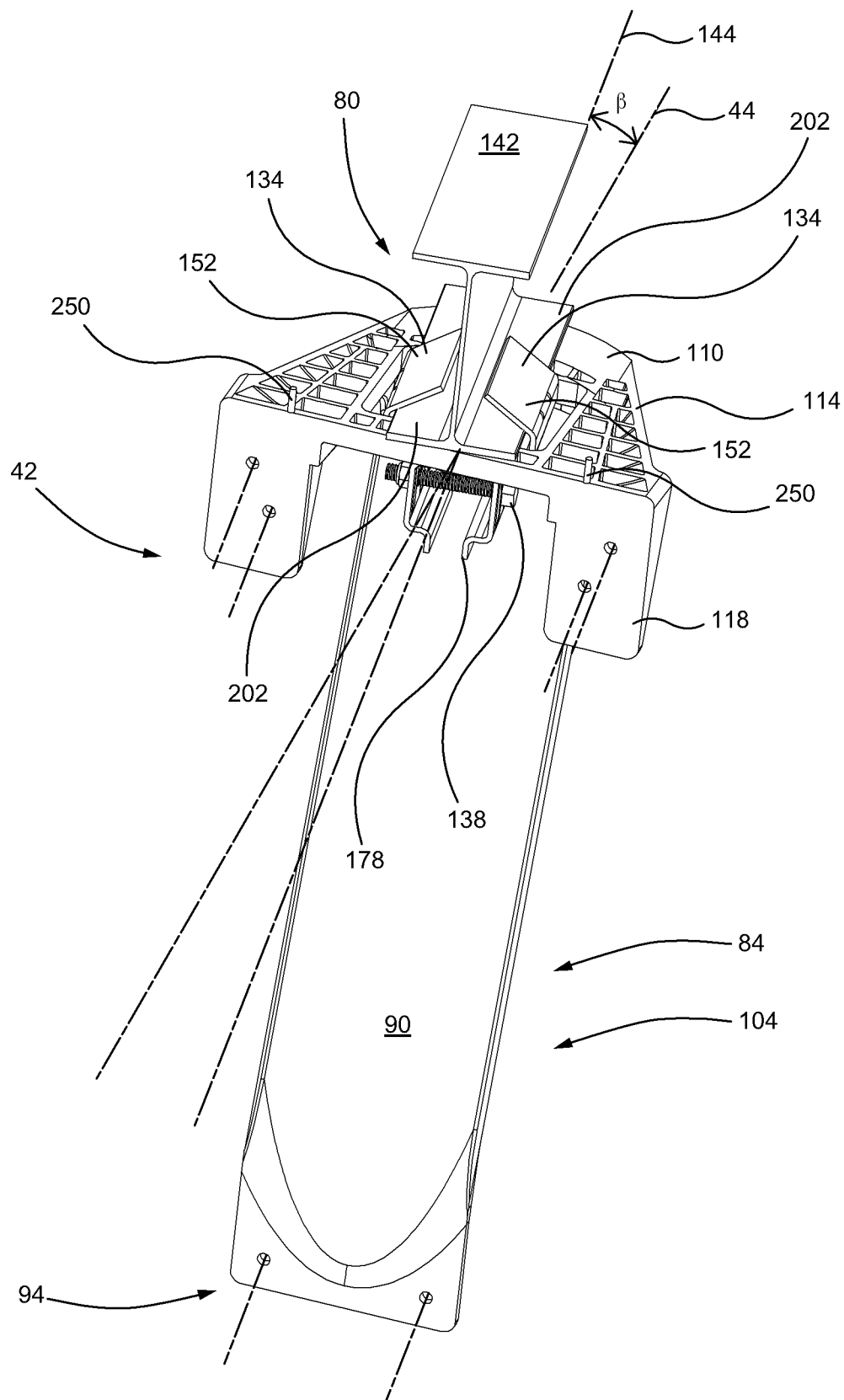


FIG. 23

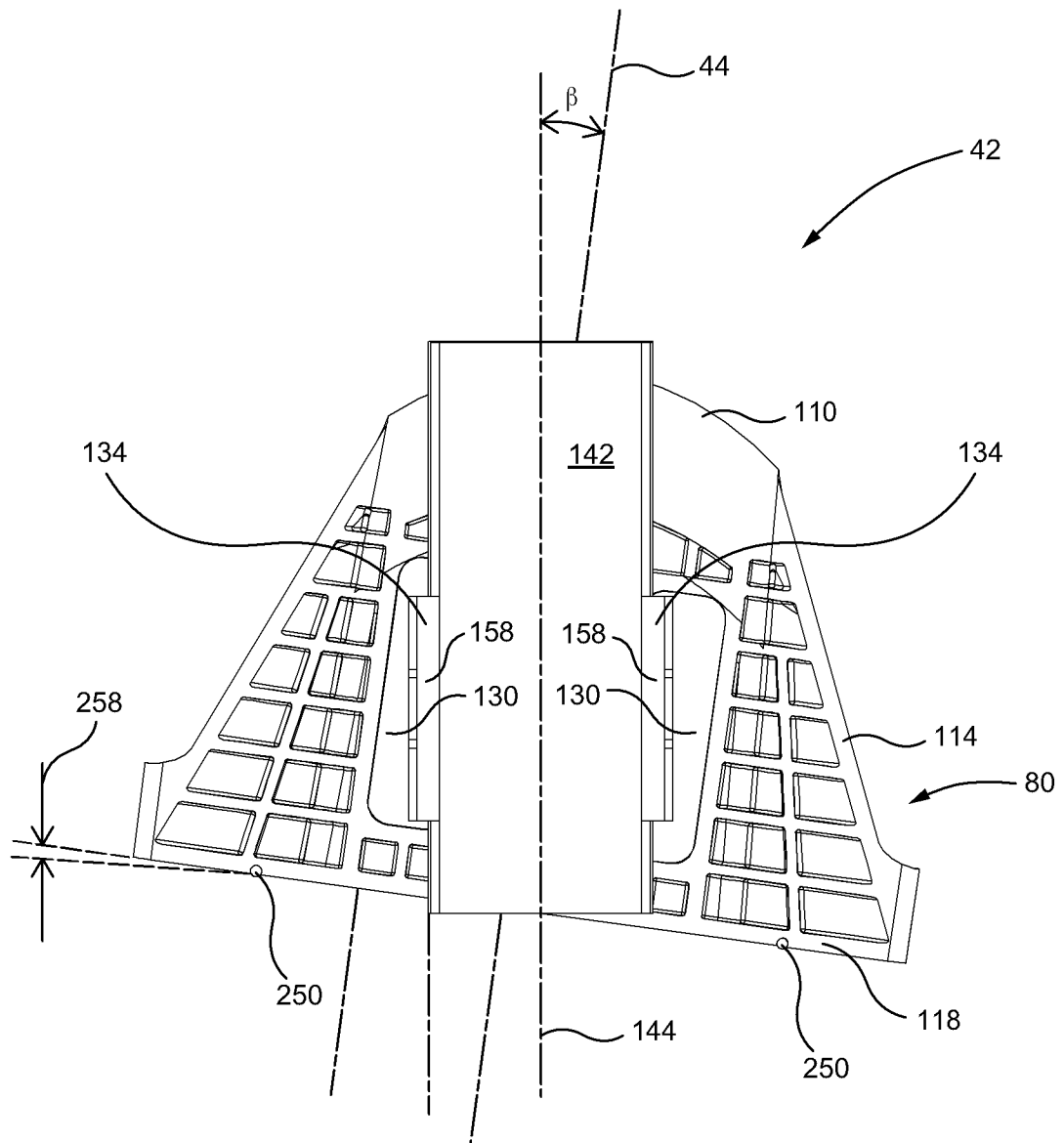


FIG. 24

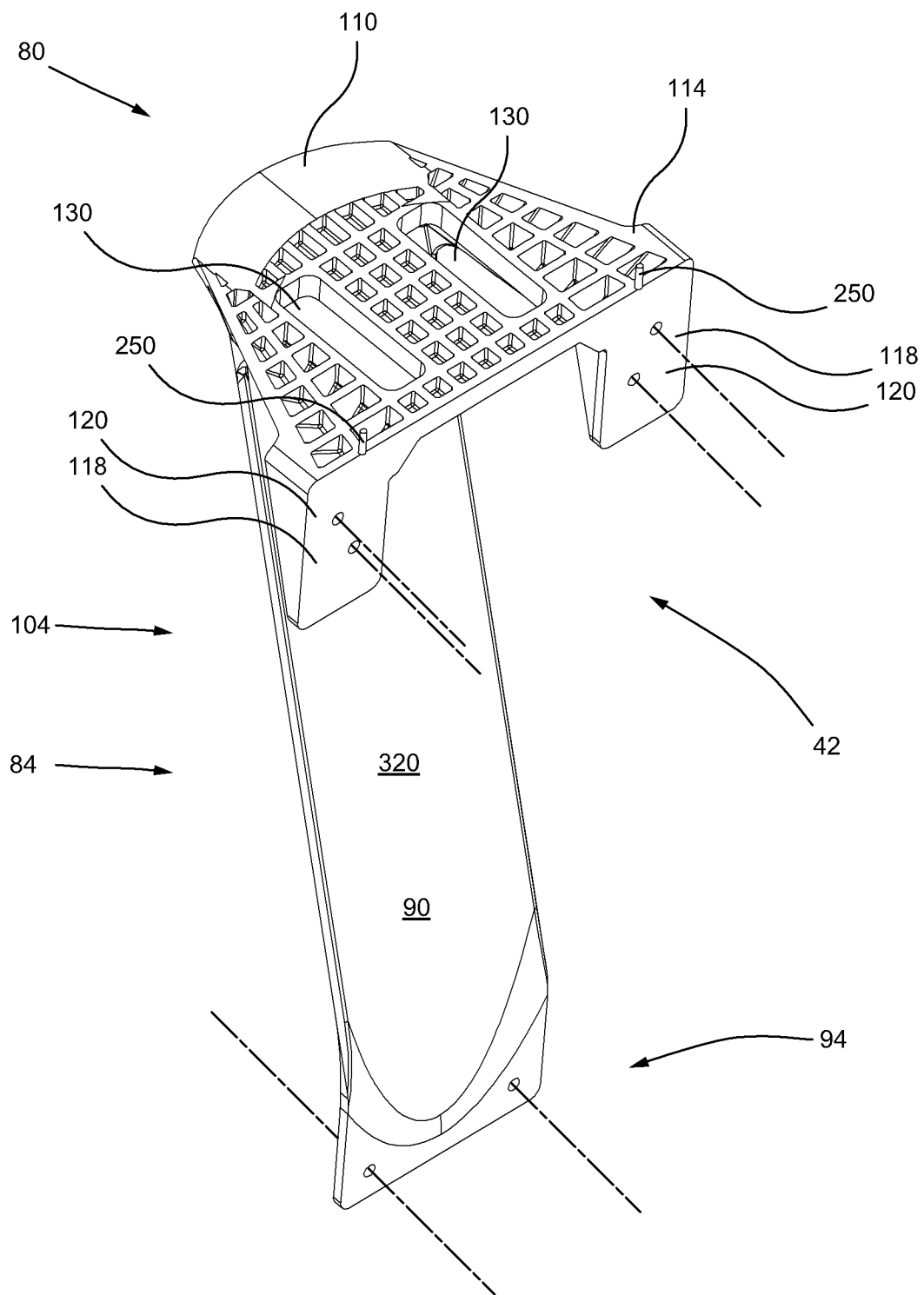


FIG. 25

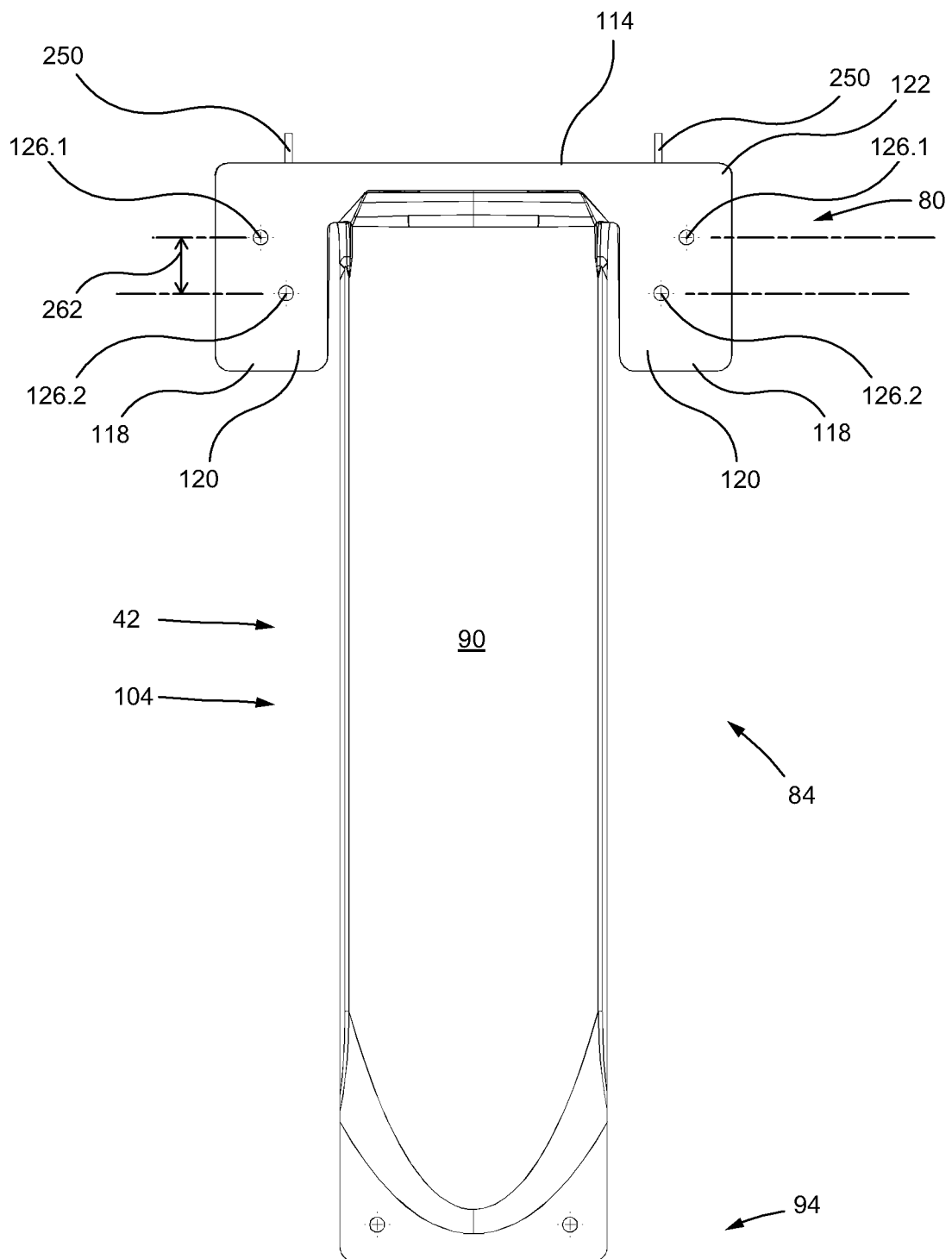


FIG. 26

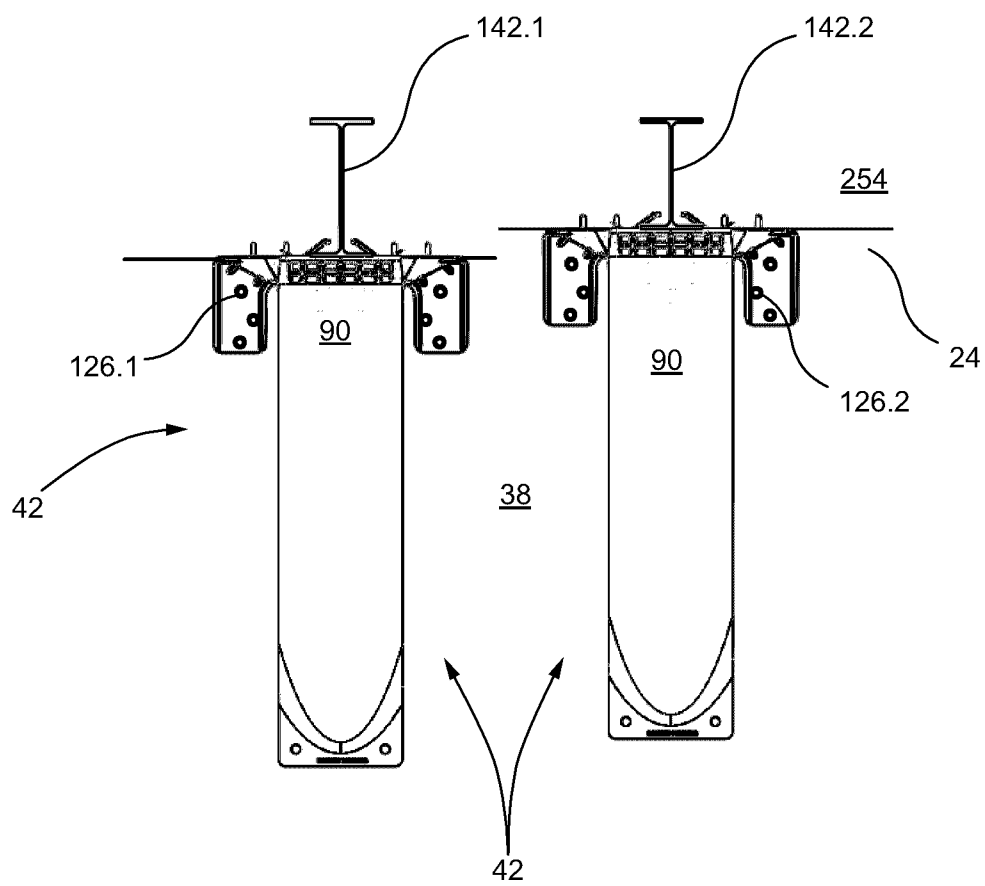


FIG. 27

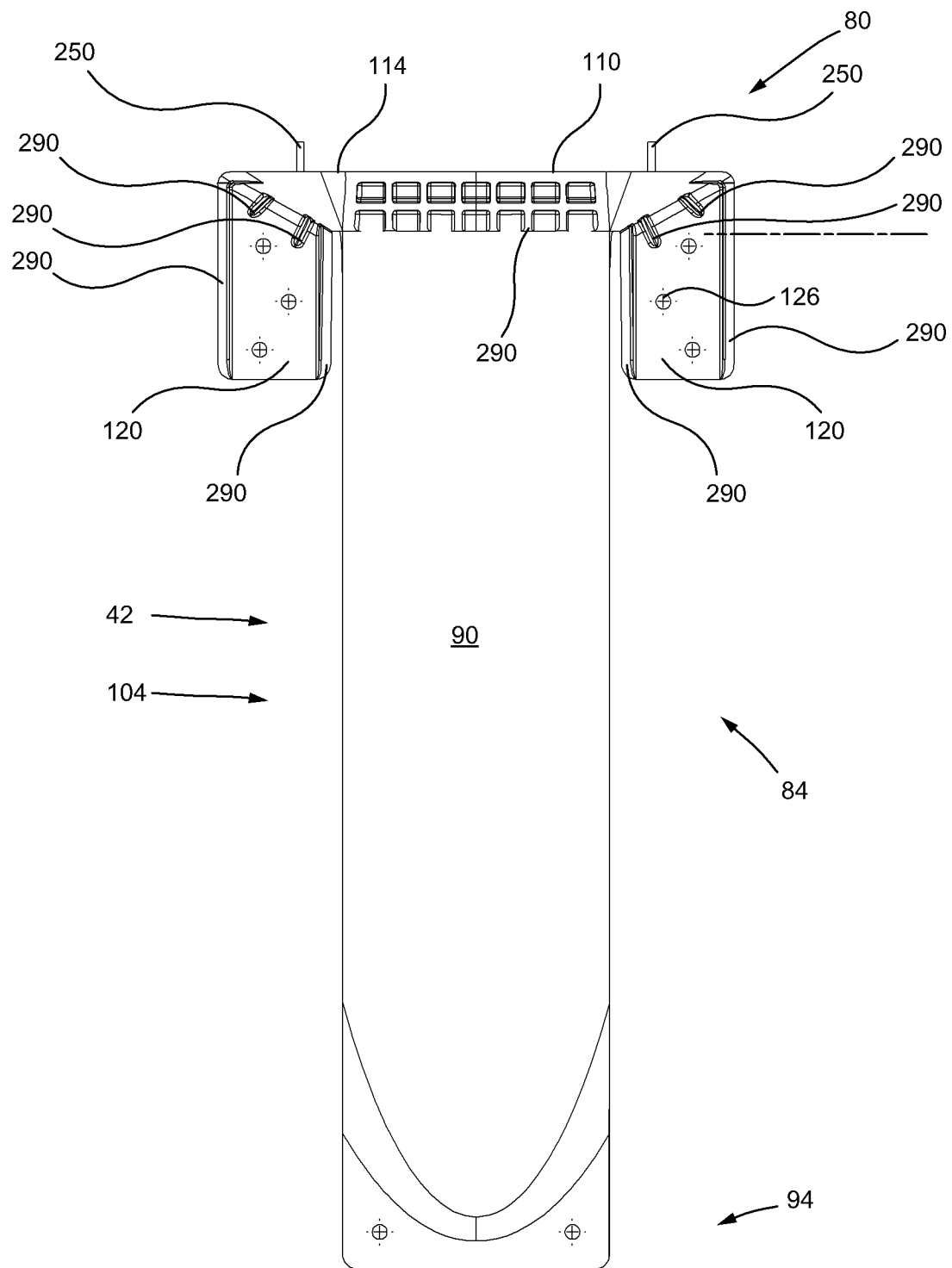


FIG. 28





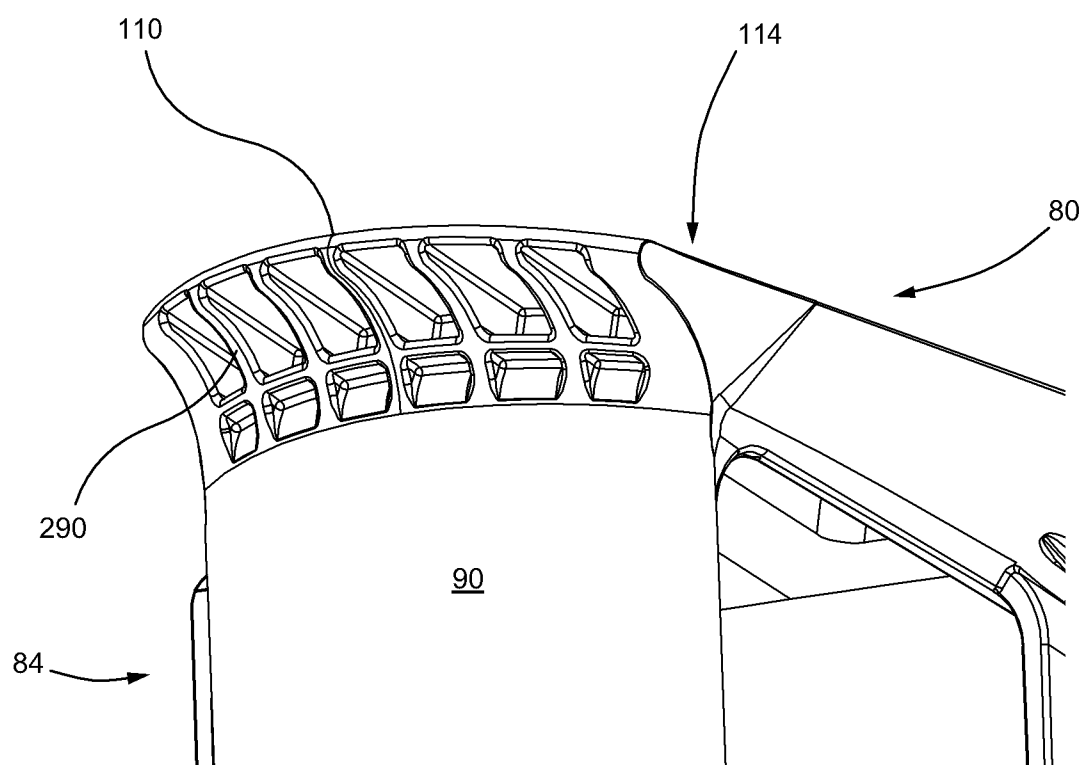


FIG. 31

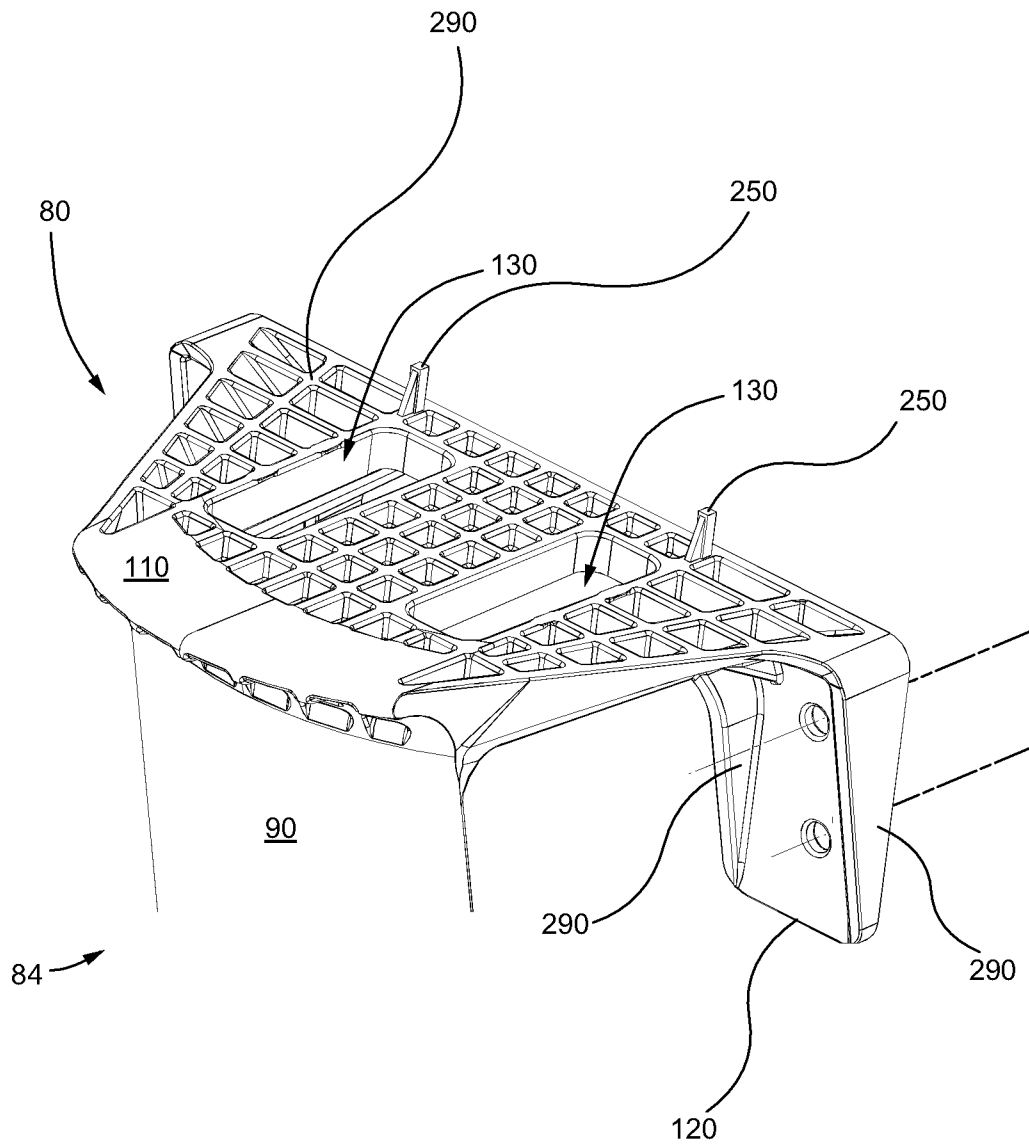


FIG. 32

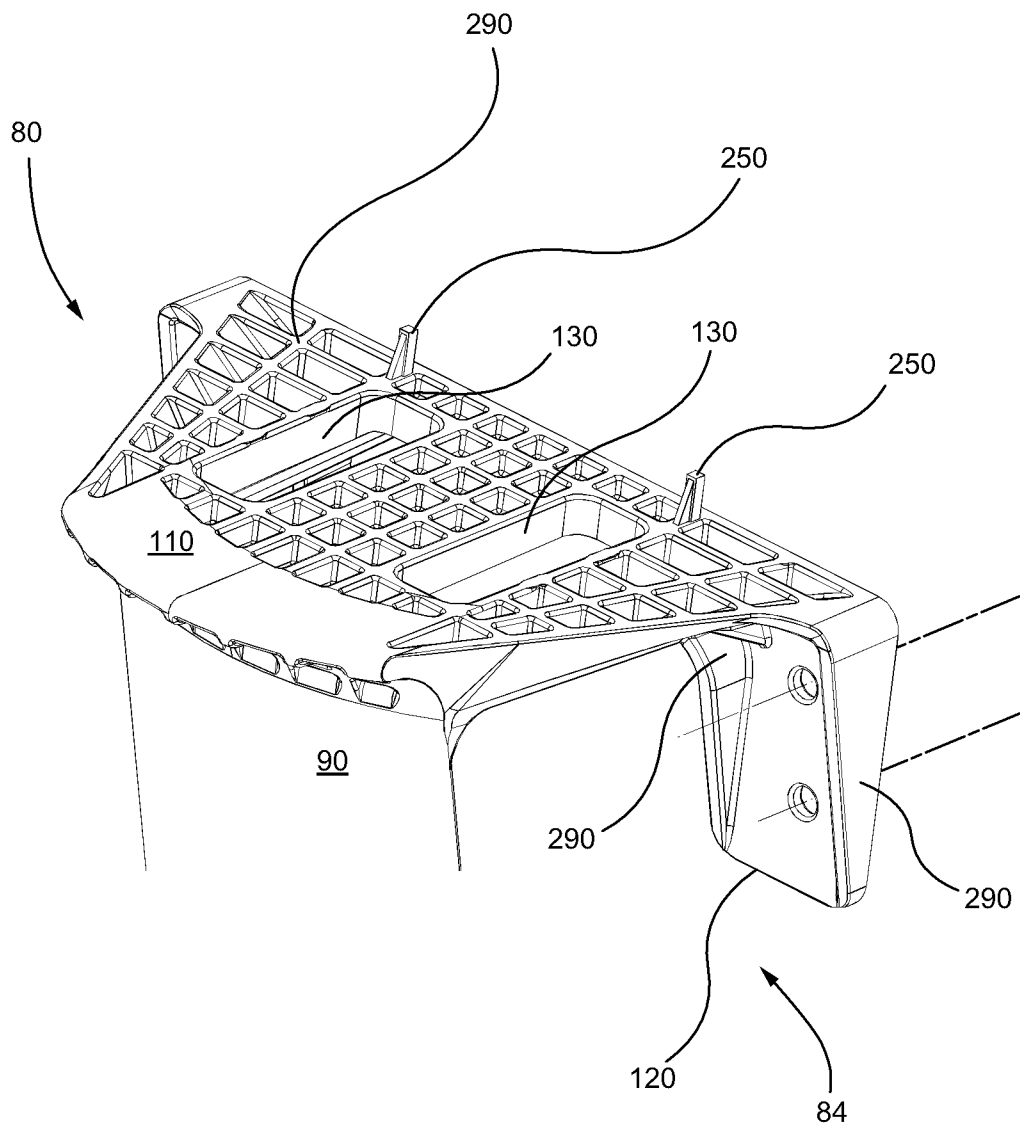


FIG. 33

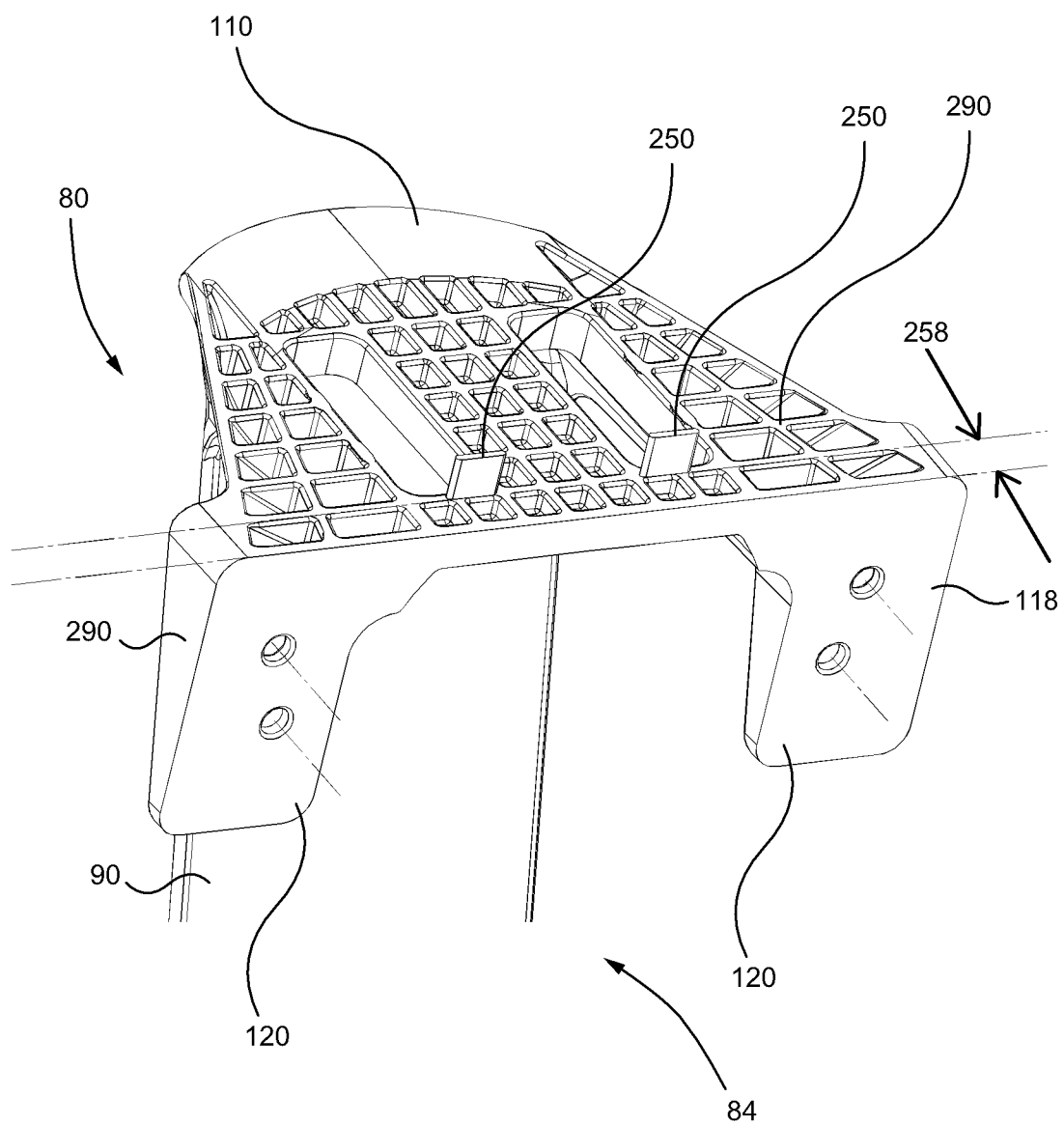


FIG. 34

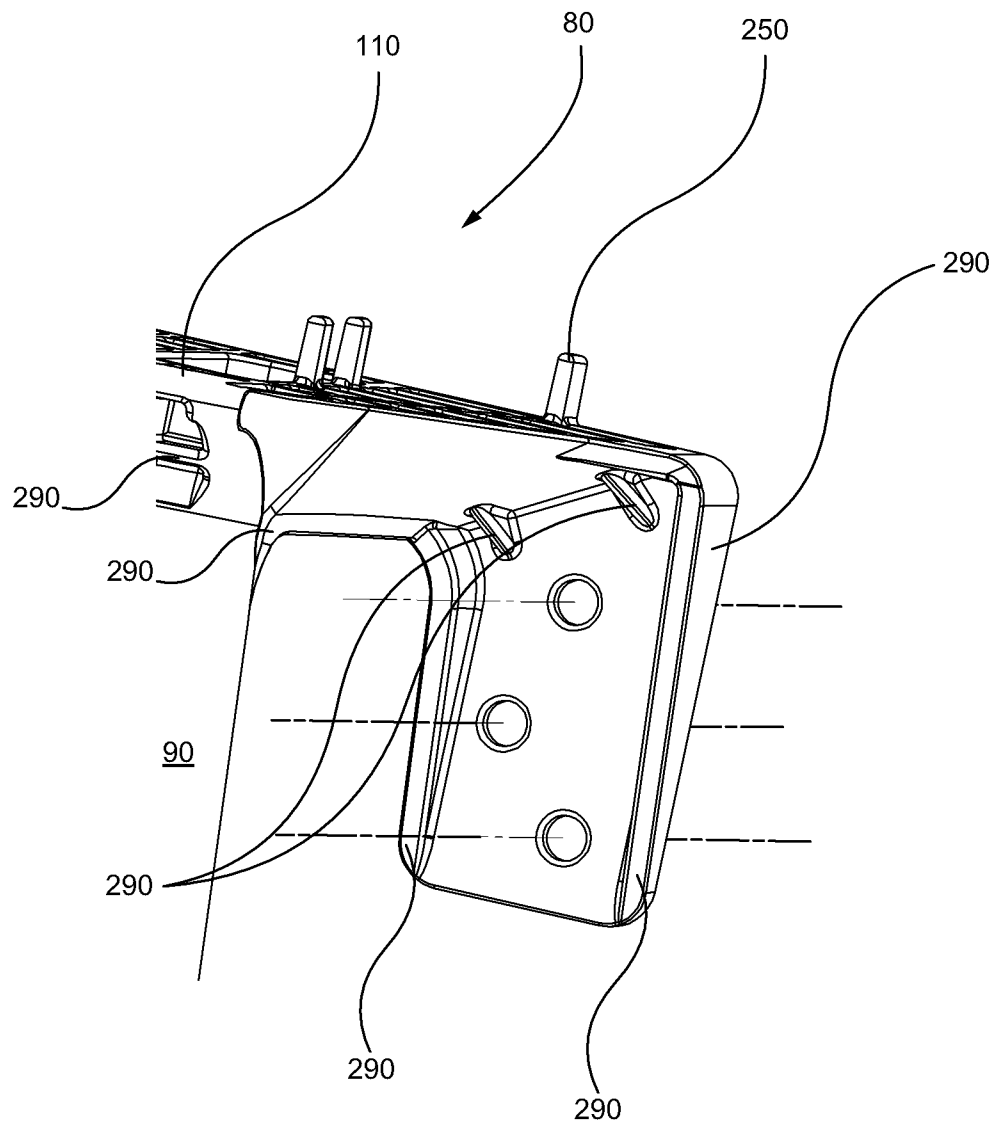


FIG. 35

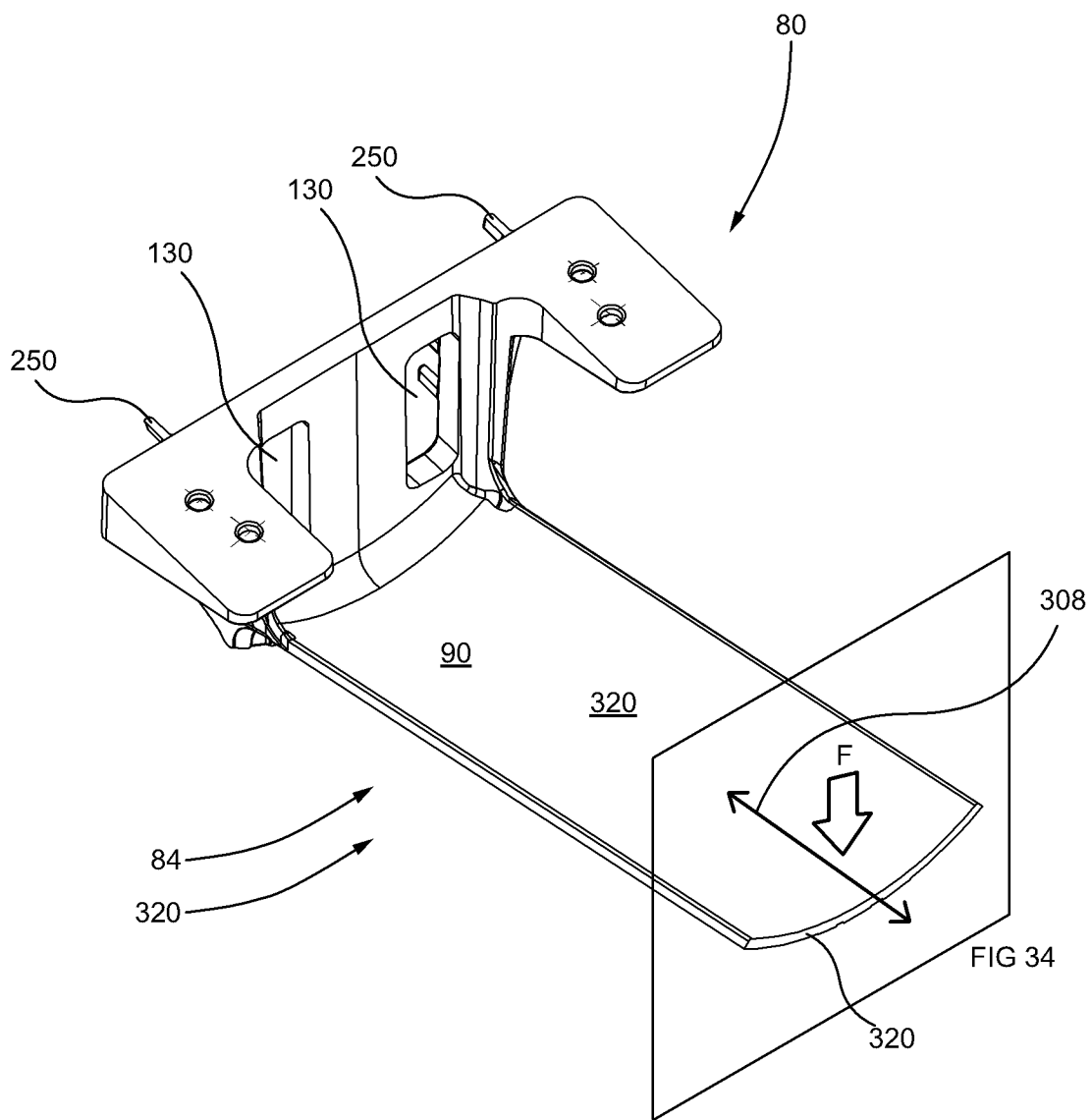
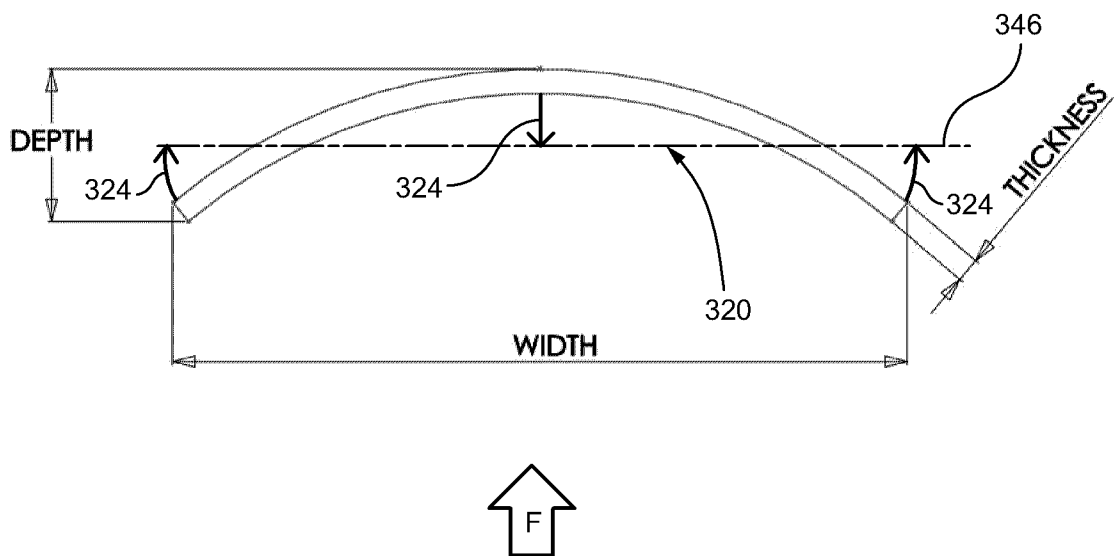


FIG. 36



(SECTION OF PLANE OF FIG. 33)

FIG. 37

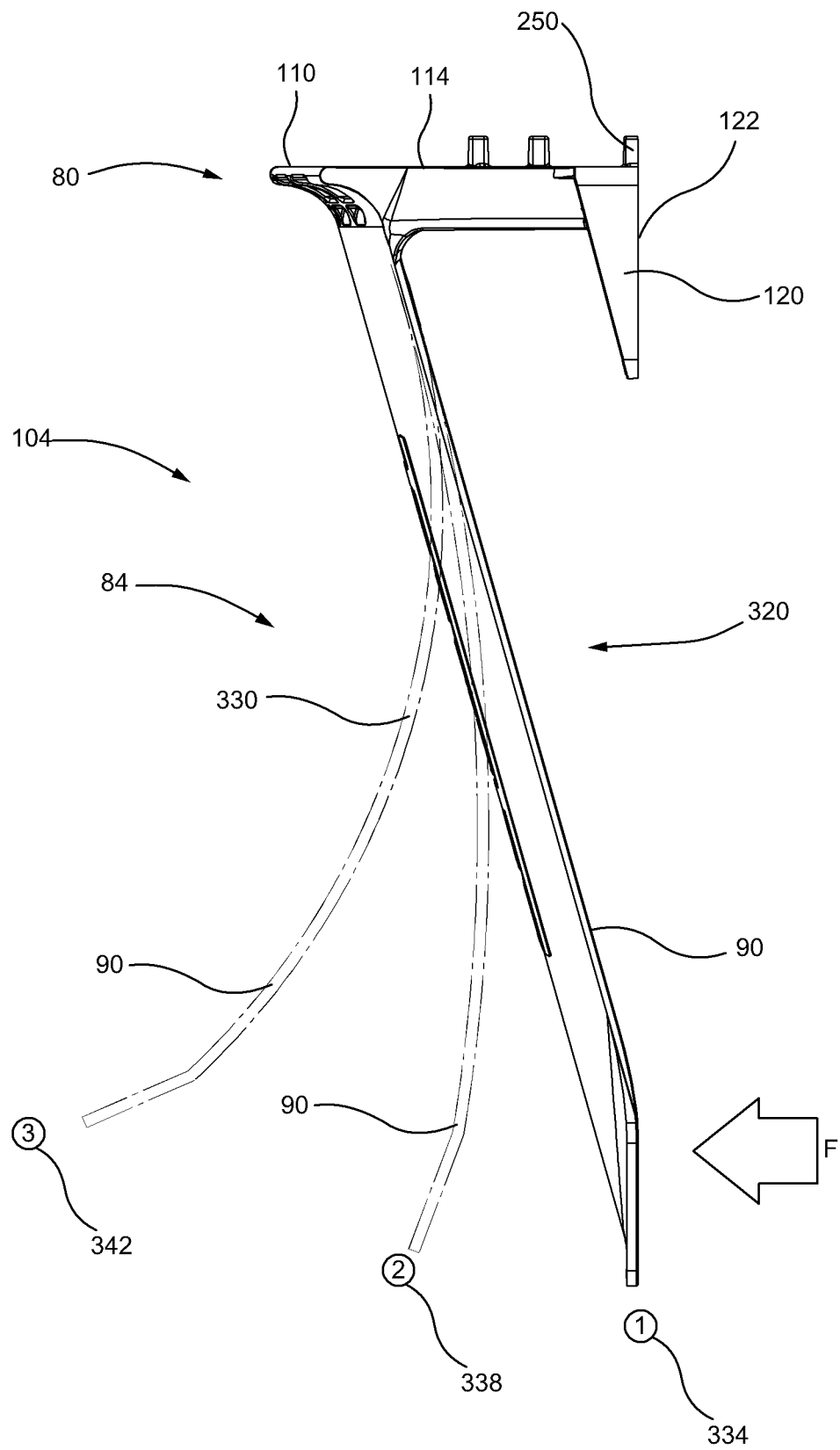


FIG. 38

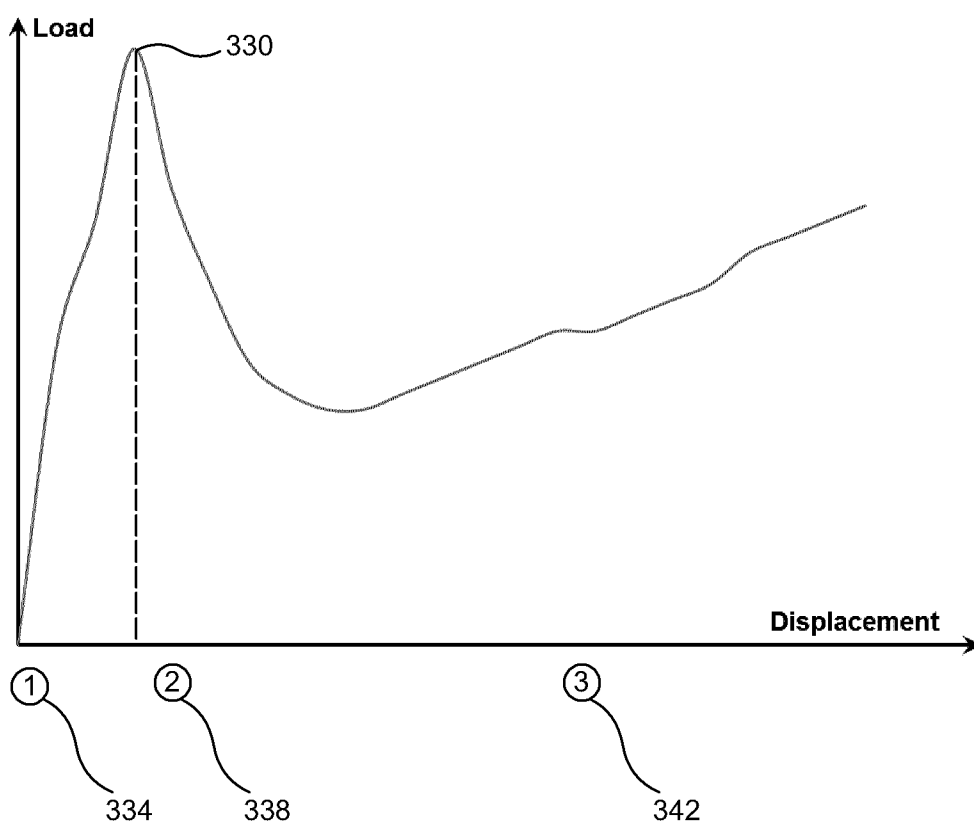


FIG. 39

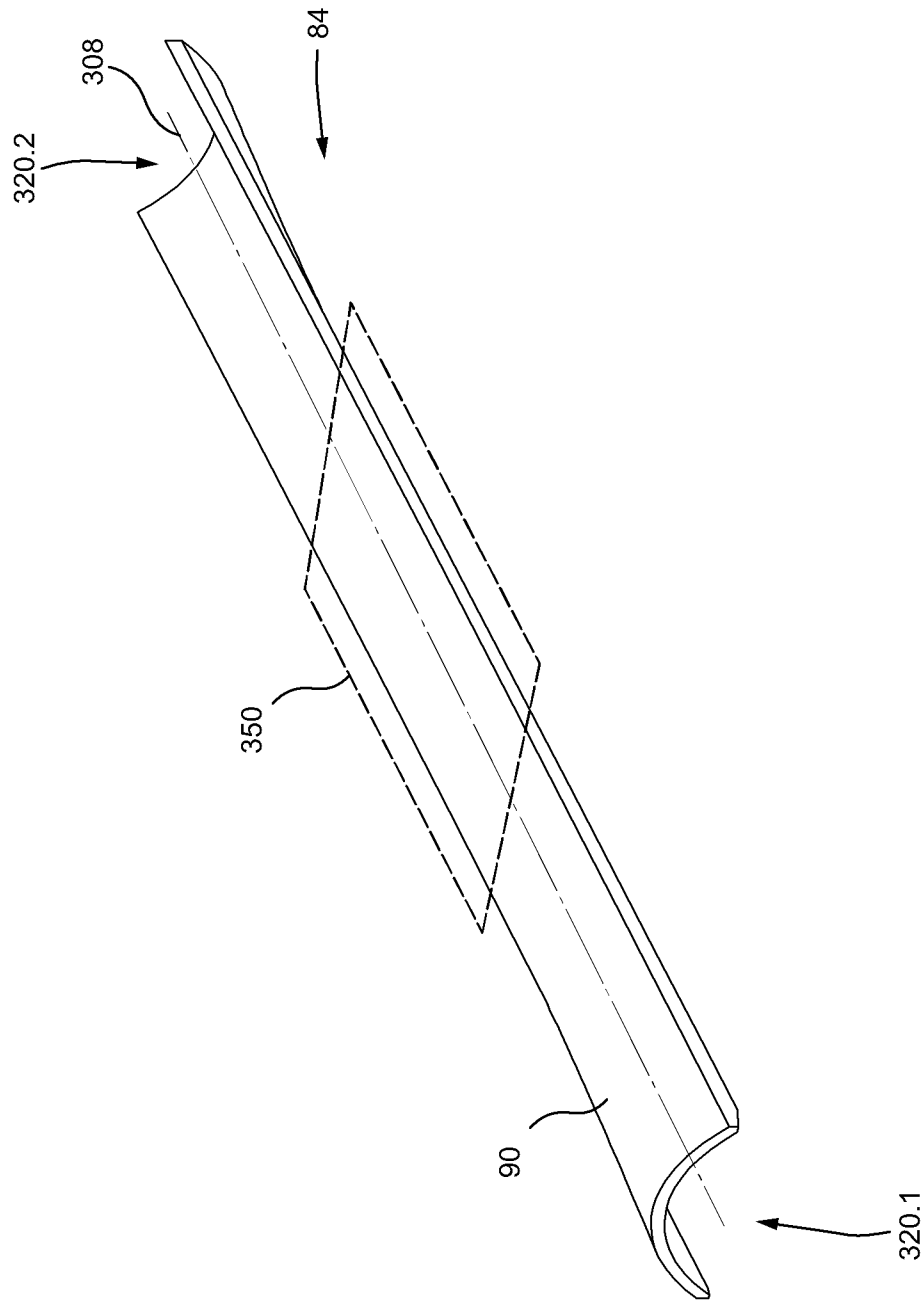


FIG. 40

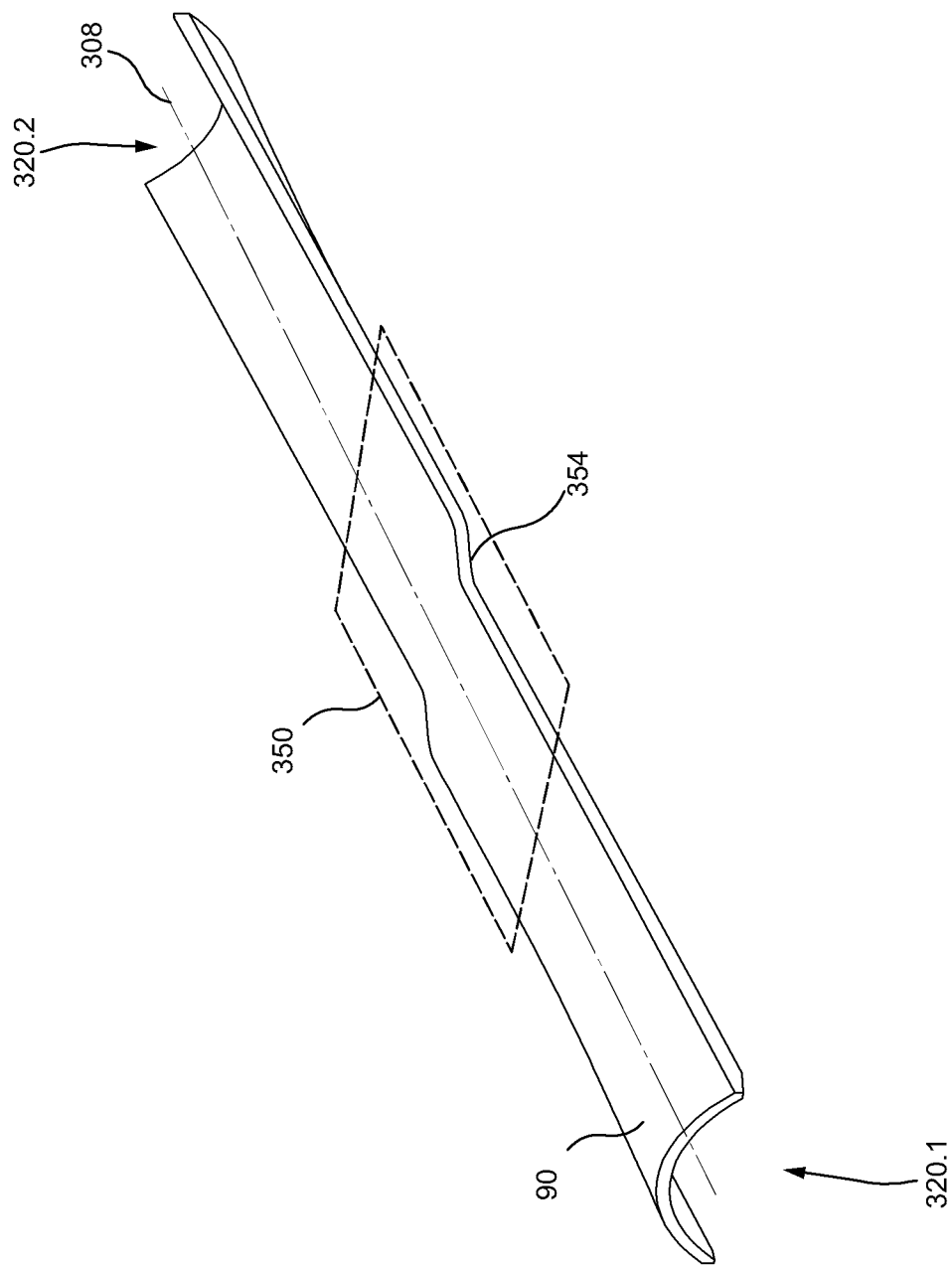


FIG. 41