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(54) **INDOOR UNIT FOR AIR CONDITIONER, AND AIR CONDITIONER**

INNENRAUMEINHEIT FÜR KLIMAANLAGE UND KLIMAANLAGE

UNITÉ INTÉRIEURE POUR CLIMATISEUR, ET CLIMATISEUR

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(56) References cited:

EP-A2- 2 568 226 **JP-A- 2006 177 604**
JP-A- 2012 037 085 **JP-A- 2013 134 025**
JP-A- 2015 049 008 **JP-A- 2015 105 771**
JP-B2- 3 045 159 **JP-B2- 5 803 898**
KR-A- 20070 053 549

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Description

Technical Field

[0001] The present invention relates to a structure of a heat exchanger of an indoor unit for an air-conditioning apparatus, and an air-conditioning apparatus including the indoor unit for an air-conditioning apparatus.

Background Art

[0002] A conventional indoor unit for an air-conditioning apparatus includes devices such as a heat exchanger, a fan, and an air-flow-direction control plate, and a box-shaped casing that houses these devices. The indoor unit allows refrigerant to circulate between the indoor unit and an outdoor unit that are connected to each other by pipes. The heat exchanger causes air flowing through the heat exchanger and the refrigerant flowing through the heat exchanger to reject heat or remove heat therebetween, thereby cooling or heating the air. The cooled or heated air is blown out from an air outlet, to adjust the temperature of air inside a room. For such an indoor unit for an air-conditioning apparatus, as a structure that improves the performance of an air-conditioning apparatus by increasing heat-rejecting efficiency or heat-removing efficiency, a structure has been proposed in which a propeller fan is arranged on the windward side of the heat exchanger.

[0003] According to an indoor unit for an air-conditioning apparatus disclosed in Patent Literature 1, for example, a propeller fan is arranged on the upstream side of the heat exchanger, and an air outlet is provided in a lower portion of the casing. The heat exchanger includes a single or a plurality of heat exchanger blocks. The air is supplied to the heat exchanger by the propeller fan, and the air that is heat-exchanged and conditioned is blown out from the air outlet.

[0004] According to an indoor unit for an air-conditioning apparatus disclosed in Patent Literature 2, a propeller fan is arranged on the upstream side of the heat exchanger, and an air outlet is provided in a lower portion of the casing. The heat exchanger includes a plurality of heat exchanger blocks, and the heat exchanger blocks are arranged in a reverse V shape in side view. An air mixing promoting element is provided so that the air passing through each of the heat exchanger blocks is mixed.

[0005] Patent Literature 3 describes an air conditioner which includes: a front-surface heat exchanger; a back-surface heat exchanger; and a drain pan arranged below the front-surface heat exchanger, with an air diffuser provided below the drain pan. At a time of a dehumidification operation, an evaporation range is arranged in a front-surface side portion of the front-surface heat exchanger, and a liquid inlet of the front-surface heat exchanger is provided in a lower portion of the front-surface side portion of the front-surface heat exchanger and in a heat transfer tube above a lowermost heat transfer tube in the

front-surface side portion of the front-surface heat exchanger.

[0006] Patent Literature 4 describes an indoor unit which includes a casing having an air inlet in an upper portion thereof and an air outlet in a lower portion of a front face thereof, a fan, a heat exchanger, and a louver configured to redirect air blown out of the air outlet in a vertical direction. The louver includes a main louver and a sub-louver.

[0007] Patent Literature 5 describes an indoor unit of an air conditioner which includes: a cabinet forming an internal space; an indoor heat exchanger provided inside the cabinet for heat exchange of the sucked air, and an indoor fan provided inside the cabinet to suck outside air.

[0008] Patent Literature 6 describes an indoor machine of an air conditioner which exhibits an excellent amenity, enables an effective utilization of the indoor space, gives a least oppressive feeling to a person staying in a room.

Citation List

Patent Literature

[0009]

Patent Literature 1: WO2010/089920

Patent Literature 2: WO2016/002015

Patent Literature 3: JP 5 803898 B2

Patent Literature 4: EP 2 568 226 A2

Patent Literature 5: KR 2007 0053549 A

Patent Literature 6: JP 3 045159 B2

Summary of Invention

Technical Problem

[0010] However, in the indoor unit for an air-conditioning apparatus disclosed in Patent Literature 1, since the propeller fan is arranged on the windward side of the heat exchanger, the air that has passed through the heat exchanger is not mixed. Therefore, the variation in blow-off temperature and humidity distribution increases due to the air velocity difference of blown-out air at the air outlet of the indoor unit during cooling operation. When a plurality of refrigerant flow passages and a plurality of refrigerant outlets are provided in the heat exchanger during the cooling operation, the refrigerant at any one of the plurality of refrigerant outlets is brought in a dry state due to the drift of the refrigerant and the difference in the heat load for each refrigerant flow passage, resulting in an increase in temperature of the refrigerant. In this case, the temperature and humidity distribution of the blown-out air further expands. In Patent Literature 2, the air mixing promoting element is provided in the vicinity of the heat exchanger, and the temperature and humidity distribution of the blown-out air is averaged. However, when the air-flow-direction control plate is provided in the

vicinity of the heat exchanger, the air mixing promoting element cannot be provided, resulting in dew condensation on the air-flow-direction control plate due to the influence of the wake of the heat exchanger.

[0011] The present invention has been made to solve the above-mentioned problems, and an object thereof is to provide an indoor unit for an air-conditioning unit that can suppress dew condensation on an air-flow-direction control plate in a configuration in which the air-flow-direction control plate is arranged in a vicinity of at least one of a plurality of heat exchanger blocks of a heat exchanger, and the air-conditioning unit. Solution to Problem

[0012] The above problems are solved by the subject-matter according to the independent claim. An indoor unit for an air-conditioning apparatus of the present invention includes a casing, an air inlet provided on the casing, an air outlet opening in a bottom surface of the casing, a heat exchanger arranged in an air passage extending from the air inlet to the air outlet, a fan in the air passage between the air inlet and the heat exchanger, and an air-flow-direction control plate provided in the air passage between the heat exchanger and the air outlet, wherein the heat exchanger includes a plurality of heat exchanger blocks that are arranged in a front-and-rear direction of the casing, and includes a refrigerant inlet through which refrigerant flows into the heat exchanger during a cooling operation, and a refrigerant outlet through which the refrigerant flows out of the heat exchanger, the air-flow-direction control plate is provided in the vicinity of one of the heat exchanger blocks, the refrigerant outlet is provided in a heat exchanger block other than the heat exchanger block that is provided in the vicinity of the air-flow-direction control plate, and the number of refrigerant outlets is larger than the number of refrigerant inlets. Advantageous Effects of Invention

[0013] According to the present invention, the heat exchanger of the indoor unit for an air-conditioning apparatus includes a refrigerant outlet that is provided in the heat exchanger block other than the heat exchanger block that is disposed in the vicinity of the air-flow-direction control plate. Thus, the variation in the temperature and humidity distribution of the blown-out air passing through the air-flow-direction control plate can be reduced, even when the refrigerant in the vicinity of the refrigerant outlets is in the dry state. Therefore, the dew condensation on the air-flow-direction control plate can be suppressed.

Brief Description of Drawings

[0014]

[Fig. 1] Fig. 1 is a perspective view illustrating an indoor unit for an air-conditioning apparatus according to Embodiment 1 of the present invention.

[Fig. 2] Fig. 2 is an explanatory view illustrating a cross section A-A perpendicular to a longitudinal di-

rection of the indoor unit in Fig. 1.

[Fig. 3] Fig. 3 is a view illustrating a refrigerant flow passage of a heat exchanger 1 illustrated in FIG. 2.

[Fig. 4] Fig. 4 is a view illustrating a refrigerant flow passage of a heat exchanger according to Embodiment 2 of the present invention.

[Fig. 5] Fig. 5 is a cross sectional view illustrating a heat exchanger in a modification example of the heat exchanger according to Embodiment 2 of the present invention.

Description of Embodiments

[0015] Embodiments of the present invention will be described hereinafter with reference to the drawings. In the drawings, devices denoted by the same reference symbols are the same or corresponding devices, and the same applies throughout the entire text of the specification. Further, the modes of components described in the entire text of the specification are merely illustrative, and the present invention is not limited to those described herein. In particular, combinations of the components are not limited to the combinations in embodiments, and components described in one embodiment may be applied to other embodiments. Furthermore, with regard to a plurality of devices of the same type that are distinguished by suffixes, in a case where the devices are not particularly required to be distinguished or specified, the suffixes are omitted in some cases. In addition, the relationship of sizes of the components in the drawings may differ from the actual sizes.

Embodiment 1

<Configuration of Indoor Unit 100>

[0016] Fig. 1 is a perspective view of an indoor unit 100 of an air-conditioning apparatus according to Embodiment 1 of the present invention. As illustrated in Fig. 1, the indoor unit 100 includes a casing 50 having a substantially cuboid shape. The casing 50 includes a front panel 52 on a front surface of the indoor unit 100. A surface facing the front panel 52 of the casing 50 includes a back panel 51. The indoor unit 100 has the back panel 51 that is attached and fixed to an installation wall surface of a room. Two air inlets 60 are arranged in a top surface of the casing 50. An air outlet 70 is provided in a bottom surface of the casing 50.

[0017] Fig. 2 is an explanatory view illustrating a cross section A-A perpendicular to a longitudinal direction of the indoor unit 100 of Fig. 1. The cross section A-A is a cross section taken along the center of an axial-flow fan 2. An internal structure and a flow of air in the indoor unit 100 of the air-conditioning apparatus according to Embodiment 1 will be described with reference to Fig. 2. The air inlet 60 is formed in the top surface of the casing 50, and the air outlet 70 is formed in the bottom surface of the casing 50. An air passage 55 extending from the air

inlet 60 to the air outlet 70 is formed inside the casing 50. In the air passage 55, the axial-flow fan 2 is arranged directly below the air inlet 60. The axial-flow fan 2 rotates to suck the air outside the casing 50 into the inside of the air passage 55 from the air inlet 60. A heat exchanger 1 is arranged below the axial-flow fan 2. The heat exchanger 1 includes a plurality of heat exchanger blocks 10a to 10d arranged in a W shape in cross-section perpendicular to a longitudinal direction of the casing 50. The heat exchanger 1 is arranged between the front panel 52 and the back panel 51, and the air delivered from the axial-flow fan 2 exchanges heat with refrigerant that has passed through the heat exchanger 1 and passes through a heat transfer tube 6. The air heat-exchanged by the heat exchanger 1 is sent toward the air outlet 70. Note that in Embodiment 1, the heat exchanger blocks 10a to 10d are arranged in a W shape, but the arrangement of the heat exchanger blocks 10a to 10d is not limited to its form. The number of the plurality of heat exchanger blocks 10 is not limited to four. The plurality of heat exchanger blocks 10 are arranged in a front-and-rear direction of the casing 50, and may be in a form such as an N shape, an M shape, and a V shape, for example.

[0018] A drain pan 20 is arranged below the heat exchanger 1. The drain pan 20 includes a drain pan portion 20a and a drain pan portion 20b for receiving dew condensation water attached to the heat exchanger 1. The drain pan portions 20a, 20b cover two lower apexes of the W-shape of the heat exchanger 1 from below, respectively. The air that has passed through the heat exchanger 1 is divided, thereby to flow through divided air passages formed between the drain pan portion 20a and a front air passage wall 52a of the front panel 52, between the drain pan portion 20a and the drain pan portion 20b, and between the drain pan portion 20b and a rear air passage wall 51a of the back panel 51, respectively. The air passage formed between the drain pan portion 20a and the front air passage wall 52a of the front panel 52 is referred to as a front air passage 56a. The air passage formed between the drain pan portion 20a and the drain pan portion 20b is referred to as a central air passage 56b. The air passage formed between the drain pan portion 20b and a rear air passage wall 51a of the back panel 51 is referred to as a rear air passage 56c. The front air passage 56a, the central air passage 56b, and the rear air passage 56c in Embodiment 1 correspond to the "divided air passages" of the present invention.

[0019] The air that has passed through the heat exchanger block 10a of the heat exchanger 1 mainly passes through the front air passage 56a, the heat exchanger block 10a being the closest to the front panel 52. An air-flow-direction control plate 3 is disposed in the front air passage 56a. The air-flow-direction control plate 3 is formed in a thin-plate shape, and the plate-shaped flat portion of the air-flow-direction control plate 3 is normally disposed to be in parallel to a direction of air flowing in the front air passage 56a. A plurality of air-flow-direction control plates 3 are disposed along the front air passage

56 extending in the longitudinal direction of the casing 50a. The air-flow-direction control plate 3 changes the angle of the flat portion to change the direction of air to be blown out from the air outlet 70. Note that the front air passage 56a in Embodiment 1 corresponds to a "first divided air passage" in the present invention. In addition, the heat exchanger block 10a corresponds to a "first heat exchanger block" in the present invention. That is, the divided air passage into which the air that has passed through the "first heat exchanger block" flows corresponds to the "first divided air passage," and the plurality of air-flow-direction control plates 3 are disposed in the first divided air passage.

[0020] The air that has passed through the heat exchanger blocks 10b and 10c at the center of the heat exchanger 1 mainly passes through the central air passage 56b. Baffle plates 21a, 21b are arranged in the central air passage 56b, and the air passing through the central air passage 56b is rectified by the baffle plates 21a, 21b to flow in a predetermined direction.

[0021] The air that has passed through the heat exchanger block 10d of the heat exchanger 1 mainly passes through the rear air passage 56c, the heat exchanger block 10d being the closest to the back panel 51. The rear air passage wall 51a has an upper portion that is parallel to the back surface of the casing 50, and a lower portion of the rear air passage wall 51a is formed to extend downward below the heat exchanger 1. A lower end of the rear air passage wall 51a is disposed below the apex disposed closer to the back panel among the two lower apexes of the W-shape of the heat exchanger 1. The air passing through the rear air passage 56c passes along the rear air passage 56c, thereby being rectified to flow obliquely downward toward the front of the casing 50.

[0022] The air outlet 70 is provided below the drain pan 20. The air outlet 70 is closed by an up-and-down airflow direction louver 30 closer to the front surface and an up-and-down airflow direction louver 40 closer to the back surface during operation of an operation. As illustrated in Fig. 2, during the operation, the up-and-down airflow direction louvers 30, 40 rotate about respective rotation shafts 31, 41 to thereby open the air outlet 70. The airflow direction can be changed to the up-and-down direction by adjusting the angles of the up-and-down airflow direction louver 30 closer to the front surface and the up-and-down airflow direction louver 40 closer to the back surface. The up-and-down airflow direction louver 30 closer to the front surface is provided with a right-and-left airflow direction louver 35 by which the airflow direction is changed to the right-and-left direction. The right-and-left airflow direction louver 35 changes the angle to a right-and-left direction of the casing 50 to change the airflow direction.

<Structure of Heat Exchanger 1>

[0023] Fig. 3 is a view illustrating a refrigerant flow pas-

sage 80 of the heat exchanger 1 illustrated in FIG. 2. The heat exchanger 1 includes a plurality of heat exchanger blocks 10a to 10d arranged in a W shape, in a cross-section perpendicular to a longitudinal direction of the casing 50. The heat exchanger blocks 10a to 10d each include a primary heat exchange portion 4 and an auxiliary heat exchange portion 5. The auxiliary heat exchange portion 5 is arranged so as to be overlapped with the windward side of the primary heat exchange portion 4. Note that the windward side means an upstream side of the air flow generated by the rotation of the axial-flow fan 2. The leeward side means a downstream side of the air flow generated by the rotation of the axial-flow fan 2. The auxiliary heat exchange portion 5 is arranged to mainly increase a subcooled region during heating operation to thereby improve the heat exchange performance. The primary heat exchange portion 4 and the auxiliary heat exchange portion 5 each include the heat transfer tubes 6 each configured to extend linearly in the longitudinal direction of the casing 50 and be turned back at the end thereof, and fins 7 each are a thin strip-shaped metal plate. A plurality of fins 7 are arranged at fine intervals in the longitudinal direction of the casing 50, i.e., in a direction in which the heat transfer tubes 6 extend linearly. The fin 7 has holes therein through which the heat transfer tubes 6 pass, and is assembled so that the heat transfer tubes 6 pass through the holes.

[0024] The heat transfer tube 6 is turned back a plurality of number of times at the ends in the longitudinal direction of the heat exchanger 1 to form the refrigerant flow passage 80. In Embodiment 1, the primary heat exchange portion 4 includes the heat transfer tubes 6 that are arrayed in two rows on the windward side and the leeward side in a plane so that the two rows are arranged in parallel to each other, and the heat transfer tubes 6 arrayed in the plane are connected at their ends. For example, the plurality of heat transfer tubes 6 arrayed on the cross section illustrated in Fig. 3 are connected to one another at their ends by U-shaped connecting tubes. In Embodiment 1, the auxiliary heat exchange portion 5 includes the heat transfer tubes 6 that are arrayed in a single row in a plane. In Fig. 3, a dotted line connecting the adjacent heat transfer tubes 6 indicates that the adjacent heat transfer tubes 6 are connected to each other at the ends on the rear side of the heat exchanger 1 as shown in Fig. 3. In Fig. 3, a solid line connecting the adjacent heat transfer tubes 6 indicates that the adjacent heat transfer tubes 6 are connected to each other at the ends on the front side of Fig. 3 of the heat exchanger 1 as shown in Fig. 3.

[0025] In the heat exchanger 1 illustrated in Fig. 3, during a cooling operation, the auxiliary heat exchange portion 5 is located on the upstream side of the refrigerant flow passage 80 and the primary heat exchange portion 4 is located on the downstream side of the refrigerant flow passage 80. The refrigerant delivered from the outdoor unit flows into the heat transfer tube 6 from a refrigerant inlet 81 in the top of an auxiliary heat exchange

portion 5a of the heat exchanger block 10a closest to the front panel 52. The refrigerant that has flowed from the refrigerant inlet 81 passes through the heat transfer tube 6 of the auxiliary heat exchange portion 5a of the heat exchanger block 10a, and then passes sequentially through an auxiliary heat exchange portion 5b of the heat exchanger block 10b, an auxiliary heat exchange portion 5c of the heat exchanger block 10c, and an auxiliary heat exchange portion 5d of the heat exchanger block 10d. The refrigerant flow passage 80 is provided with a bifurcation portion 82 after the refrigerant flows out of the auxiliary heat exchange portion 5d. The refrigerant flowing out of the auxiliary heat exchange portion 5d branches off into two passages of a refrigerant flow passage 80a and a refrigerant flow passage 80b from the bifurcation portion 82. The refrigerant flowing in the one refrigerant flow passage 80a flows into a primary heat exchange portion 4a of the heat exchanger block 10a closest to the front panel 52. The refrigerant flowing in the other refrigerant flow passage 80b flows into a primary heat exchange portion 4b of the heat exchanger block 10b that is closer to the front panel 52 among the central heat exchanger blocks.

[0026] The refrigerant that has branched off into the refrigerant flow passage 80a flows into the primary heat exchange portion 4a of the heat exchanger block 10a. In the primary heat exchange portion 4a, the refrigerant flows into a heat transfer tube 6a that is located in the uppermost position among the heat transfer tubes 6 arrayed on the windward side. The primary heat exchange portion 4a includes the heat transfer tubes 6 that are arrayed in two rows on the windward side and the leeward side. The refrigerant that has flowed into the primary heat exchange portion 4a passes through the row of the heat transfer tubes 6 on the windward side and the row of the heat transfer tubes 6 on the leeward side, and then flows out of the primary heat exchange portion 4a. The refrigerant that has flowed out of the primary heat exchange portion 4a flows into a primary heat exchange portion 4c of the heat exchanger block 10c. The refrigerant that has flowed into the primary heat exchange portion 4c of the heat exchanger block 10c flows into the heat transfer tube 6 that is located in the uppermost position on the windward side of the primary heat exchange portion 4c. In the primary heat exchange portion 4a, the refrigerant flows into the heat transfer tube 6 that is located in the uppermost position on the leeward side after the refrigerant has passed through the second heat transfer tube 6 from top on the windward side, and then the refrigerant flows into the third heat transfer tube 6 from the top on the windward side after the refrigerant has passed through the second heat transfer tube 6 from top on the leeward side. Then, the refrigerant passes through the heat transfer tubes 6 that are located below the third heat transfer tube 6 from the top on the windward side of the primary heat exchange portion 4c, and then flows out of the heat transfer tube 6 that is located in the lowermost position on the windward side of the primary heat ex-

change portion 4c. Then, the refrigerant flows into the primary heat exchange portion 4d of the heat exchanger block 10d closest to the back panel 51. The refrigerant that has flowed into the primary heat exchange portion 4d flows into the heat transfer tube 6 that is located in the lowermost position on the windward side, passes through the heat transfer tubes 6 that are located in a lower portion on the windward side, flows into the row of the heat transfer tubes 6 on the leeward side, and then flows out of a refrigerant outlet 83 that is provided at the center of the heat transfer tube 6 on the leeward side.

[0027] The refrigerant that has branched off into the refrigerant flow passage 80b from the bifurcation portion 82 flows into the primary heat exchange portion 4b of the heat exchanger block 10b. In the primary heat exchange portion 4b, the refrigerant flows into the heat transfer tube 6b that is located in the uppermost position among the heat transfer tubes 6 arrayed on the windward side. The primary heat exchange portion 4b includes the heat transfer tubes 6 that are arrayed in two rows on the windward side and the leeward side. The refrigerant that has flowed into the primary heat exchange portion 4b passes through the row of the heat transfer tubes 6 on the windward side and the row of the heat transfer tubes 6 on the leeward side, and then flows out of the primary heat exchange portion 4b. The refrigerant that has flowed out of the primary heat exchange portion 4b flows into the primary heat exchange portion 4d of the heat exchanger block 10d. The refrigerant that has flowed into the primary heat exchange portion 4d of the heat exchanger block 10d flows into the heat transfer tube 6 that is located in the uppermost position on the windward side of the primary heat exchange portion 4d. The refrigerant that has flowed into the heat transfer tube 6 located in the uppermost position on the windward side of the primary heat exchange portion 4d flows into the heat transfer tube 6 that is located in the uppermost position on the leeward side after the refrigerant has passed through the second heat transfer tube 6 from top on the windward side, and then the refrigerant flows into the third heat transfer tube 6 from the top on the windward side after the refrigerant has passed through the second heat transfer tube 6 from top on the leeward side. Then, the refrigerant passes through the third and fourth heat transfer tubes 6 from the top on the windward side of the primary heat exchange portion 4d, and then flows out of the primary heat exchange portion 4d. Then, the refrigerant that has flowed out of the primary heat exchange portion 4d flows into the third heat transfer tube 6 from the top on the leeward side of the primary heat exchange portion 4c. Then, the refrigerant flows from the third heat transfer tube 6 on the leeward side of the primary heat exchange portion 4c to the heat transfer tube 6 on the lowermost position, and flows out of the primary heat exchange portion 4c. The refrigerant that has flowed out of the primary heat exchange portion 4c flows into the heat transfer tube 6 located in the lowermost position on the leeward side of the primary heat exchange portion 4d of the heat ex-

changer block 10d closest to the back panel 51. The refrigerant that has flowed into the primary heat exchange portion 4d flows into the heat transfer tube 6 located in the lowermost position on the windward side, passes through the heat transfer tubes 6 that are located in a lower portion on the windward side, is transferred to the heat transfer tube 6 on the leeward side, and then flows out of a refrigerant outlet 84.

[0028] As described above, during the cooling operation, the refrigerant flowing into the heat exchanger 1 flows into the heat exchanger 1 from a single refrigerant circuit, and the refrigerant flow passage 80 branches off midway into two refrigerant circuits of the refrigerant flow passage 80a and the refrigerant flow passage 80b, and flows out of the refrigerant outlet 83 and the refrigerant outlet 84. Here, the two refrigerant outlets 83, 84 each are connected to any one of the heat transfer tubes 6 in the row on the most leeward side of the heat transfer tubes 6 in the heat exchanger block 10d closest to the back panel 51 of the heat exchanger blocks included in the primary heat exchange portion 4 and the auxiliary heat exchange portion 5 of the heat exchanger 1.

[0029] Since the drift of the refrigerant occurs in the refrigerant flow passage 80 of the heat exchanger 1 and the difference in heat load is generated for each portion of the refrigerant flow passage 80, the refrigerant may be dried in the refrigerant outlet 83 and the refrigerant outlet 84. Therefore, this may cause increase in the variation in the temperature and humidity distribution of the air that passes through the heat exchanger 1 and is blown into the air-flow-direction control plate 3. However, the refrigerant outlet 83 and the refrigerant outlet 84 through which the refrigerant flows out of the heat exchanger 1 are not arranged in the heat exchanger block 10a during the cooling operation, the heat exchanger block 10a being disposed on the windward side of the air-flow-direction control plate 3, thereby enabling the air-flow-direction control plate 3 to be arranged at a position near the heat exchanger 1 affected by the wake of the heat transfer tubes. In Embodiment 1, in the front air passage 56a into which the air that has passed through the heat exchanger block 10a flows, the variation in the temperature and humidity distribution of the air that passes through the air-flow-direction control plate 3 is not increased. Therefore, dew condensation on the air-flow-direction control plate 3 during the cooling operation can be suppressed.

[0030] To prevent dew from being condensed on the air-flow-direction control plate 3, the heat exchanger block 10 in which the refrigerant outlet 83 and the refrigerant outlet 84 are arranged does not have to be the heat exchanger block 10d closest to the back panel for cooling operation, and may be the heat exchanger block 10b or 10c that is not disposed in the vicinity of the air-flow-direction control plate 3, for example. This is because the air that has passed through the heat exchanger block 10b and the heat exchanger block 10c mainly passes through the central air passage 56b, and therefore the variation in the quality of the refrigerant in the refrigerant

outlet 83 and the refrigerant outlet 84 has less effect on the air-flow-direction control plate 3 that is disposed in the front air passage 56a. Furthermore, a detector may be provided to detect the quality of the refrigerant in the refrigerant outlet 83 and the refrigerant outlet 84. For example, the detector may detect the temperatures of pipes at the refrigerant outlet 83 and the refrigerant outlet 84.

[0031] The indoor unit 100 of an air-conditioning apparatus in which the heat exchanger 1 is provided is connected with an outdoor unit. The outdoor unit is provided with a compressor and an outdoor heat exchanger. The indoor unit 100 and the outdoor unit are connected to each other by a connection pipe through which the refrigerant flows to form a refrigeration cycle circuit.

<Effect of Embodiment 1>

[0032]

(1) The indoor unit 100 of an air-conditioning apparatus according to Embodiment 1 includes the casing 50, the air inlet 60 provided on the casing 50, the air outlet 70 opening in a bottom surface of the casing 50, the heat exchanger 1 arranged in the air passage 55 extending from the air inlet 60 to the air outlet 70, the axial-flow fan 2 arranged on the windward side of the heat exchanger 1 in the air passage 55, and the air-flow-direction control plate 3 provided in the air passage 55 between the heat exchanger 1 and the air outlet 70. The heat exchanger 1 includes a plurality of heat exchanger blocks 10 that are arranged in the front-and-rear direction of the casing, and includes the refrigerant inlet 81 through which the refrigerant flows into the heat exchanger 1, and the refrigerant outlets 83, 84 through which the refrigerant flows out of the heat exchanger 1. The air-flow-direction control plate 3 is provided in the vicinity of one of the heat exchanger blocks 10, the refrigerant outlets 83, 84 are provided in the heat exchanger block 10d other than the heat exchanger block 10a that is provided in the vicinity of the air-flow-direction control plate, and the number of refrigerant outlets 83, 84 is larger than the number of refrigerant inlets 81.

Such a configuration can suppress condensation of dew on the air-flow-direction control plate 3 even when the air-flow-direction control plate 3 is provided in the vicinity of the heat exchanger block 10a. When components such as the air-flow-direction control plate 3 are arranged in the vicinity of the heat exchanger block 10, dew concentration may be normally caused by differences in temperature and humidity of the air that has passed through each portion of the heat exchanger block 10a. The variation in temperature and humidity distribution of the air that has passed through the heat exchanger block 10d provided with the refrigerant outlets 83, 84 increases more largely than that of the air that has passed

through the other heat exchanger blocks 10 of the heat exchanger 1, but the air that has passed through the heat exchanger block 10d does not pass through the air-flow-direction control plate 3. Since the air-flow-direction control plate 3 is arranged in the vicinity of the heat exchanger block 10a, the variation in temperature and humidity of the air in contact with the air-flow-direction control plate 3 is relatively small, and hence, the dew is hardly condensed on the air-flow-direction control plate 3. Note that in Embodiment 1, the heat exchanger block 10 provided in the vicinity of the air-flow-direction control plate 3 is the heat exchanger block 10a, and the heat exchanger block 10 provided with the refrigerant outlets 83, 84 is the heat exchanger block 10d, but the present invention is not limited to the embodiment. The heat exchanger block 10 provided in the vicinity of the air-flow-direction control plate 3 and the heat exchanger block 10 provided with the refrigerant outlets 83, 84 may be different from each other.

(2) In the indoor unit 100 of an air-conditioning apparatus according to Embodiment 1, the air passage 55 is divided into a plurality of divided air passages on the downstream side of the heat exchanger 1. The air that has passed through the heat exchanger block 10a provided with no refrigerant outlets 83, 84, among the plurality of heat exchanger blocks 10, flows into the front air passage 56a that is one of the divided air passages, and the air-flow-direction control plate 3 is disposed in the front air passage 56a. Note that the heat exchanger block 10a in Embodiment 1 corresponds to a "first heat exchanger block" of the present invention, and the front air passage 56a in Embodiment 1 corresponds to a "first divided air passage" of the present invention.

With such a configuration, the air that has passed through the heat exchanger block 10a provided with no refrigerant outlets 83, 84 passes through the front air passage 56a in which the air-flow-direction control plate 3 is disposed, and therefore the air-flow-direction control plate 3 is arranged so that dew is hardly condensed on the air-flow-direction control plate 3. Note that the "first heat exchanger block" of the present invention is not limited to the heat exchanger block 10a, and may be the other heat exchanger block 10 provided with no refrigerant outlets 83, 84 among the plurality of heat exchanger blocks 10. In this case, the "first divided air passage" is not limited to the front air passage 56a. When the "first heat exchanger block" corresponds to the heat exchanger block 10b or 10c, the "first divided air passage" corresponds to the central air passage 56b. When the "first heat exchanger block" corresponds to the heat exchanger block 10d, the "first divided air passage" corresponds to the rear air passage 56c.

(3) In the indoor unit 100 of an air-conditioning apparatus according to Embodiment 1, the heat ex-

changer block 10 includes the auxiliary heat exchange portion 5 and the primary heat exchange portion 4. The auxiliary heat exchange portion 5 is arranged so as to be overlapped on the windward side of the primary heat exchange portion 4 in the air passage 55. The refrigerant inlet 81 is provided in the auxiliary heat exchange portion 5. The refrigerant outlets 83, 84 are provided in the primary heat exchange portion 4.

(4) The indoor unit 100 of an air-conditioning apparatus according to Embodiment 1 further includes a detector for detecting the dry state of the refrigerant in the refrigerant outlets 83, 84.

With such a configuration, the dry state of the refrigerant in the refrigerant outlets 83, 84 of the heat exchanger 1 is detected to adjust the opening degree of the expansion valve of the outdoor unit, and therefore the dry state of the refrigerant outlets 83, 84 can be suppressed without sacrificing the cooling efficiency.

(5) An air-conditioning apparatus according to Embodiment 1 includes an outdoor unit for an air-conditioning apparatus that includes a compressor configured to compress the refrigerant, and the indoor unit 100 of the air-conditioning apparatus according to Embodiment 1, to thereby constitute a refrigeration cycle in which the refrigerant circulates between the outdoor unit for an air-conditioning apparatus and the indoor unit 100 of the air-conditioning apparatus.

[0033] With such a configuration, even when the drift of the refrigerant occurs in the refrigerant flow passage 80 of the heat exchanger 1 of the indoor unit 100, the indoor unit 100 includes the configurations (1) to (4) described above, thereby being capable of suppressing dew condensation on the components such as the air-flow-direction control plate 3 arranged in the vicinity of the heat exchanger block 10.

Embodiment 2

[0034] An indoor unit 200 of an air-conditioning apparatus according to Embodiment 2 is obtained by modifying the structure of the refrigerant flow passage 80 of the heat exchanger 1 in the indoor unit 100 of an air-conditioning apparatus according to Embodiment 1. The following description is focused on differences between Embodiment 2 and Embodiment 1. Matters that are not particularly mentioned in Embodiment 2 are similar to those in Embodiment 1, and the same functions and components as those in Embodiment 1 are designated by the same reference signs in the following description.

[0035] Fig. 4 is a view illustrating a refrigerant flow passage 80 of a heat exchanger 201 according to Embodiment 2 of the present invention. As illustrated in Fig. 4, a refrigerant inlet 281 of the heat exchanger 201 may be arranged closer to the back panel 51, i.e., in a heat exchanger block 210d closest to the rear air passage wall.

When all of the refrigerant inlet 281 and refrigerant outlets 283, 284 of the heat exchanger 201 are arranged in the same heat exchanger block 210d closest to the rear air passage wall, a refrigerant inflow passage can be shortened between the heat exchanger 201 and a connection pipe between the indoor unit and the outdoor unit that is a flow passage through which the refrigerant flows. This enables refrigerant pressure loss to be reduced during the cooling operation, thereby improving the cooling performance of the air-conditioning apparatus. The amount of copper pipes used for the refrigerant inflow passage can be reduced by shortening the refrigerant inflow passage, thereby reducing the cost.

[0036] As illustrated in Fig. 4, the heat exchanger 201 in Embodiment 2 includes a refrigerant flow passage 280 as described below. Firstly, the refrigerant flows into the heat exchanger 201 from the refrigerant inlet 281. The refrigerant delivered from the outdoor unit flows into the heat transfer tube 6 from the refrigerant inlet 281 in the top of the auxiliary heat exchange portion 5d of the heat exchanger block 10d closest to the rear air passage wall. The refrigerant that has flowed from the refrigerant inlet 281 passes through the heat transfer tube 6 of the auxiliary heat exchange portion 5d of the heat exchanger block 10d, and then passes sequentially through the auxiliary heat exchange portion 5c of the heat exchanger block 10c, the auxiliary heat exchange portion 5b of the heat exchanger block 10b, and the auxiliary heat exchange portion 5a of the heat exchanger block 10a. A bifurcation portion 282 is provided in the heat transfer tube 6 after the auxiliary heat exchange portion 5a. The refrigerant flowing out of the auxiliary heat exchange portion 5a branches off into two passages of a refrigerant flow passage 280a and a refrigerant flow passage 280b from the bifurcation portion 282, and flows into the primary heat exchange portion 4a of the heat exchanger block 10a.

[0037] The refrigerant flowing through the refrigerant flow passage 280a flows in the row of the heat transfer tubes 6 on the windward side of the primary heat exchange portion 4a in an upward direction, flows into the row of the heat transfer tubes 6 on the leeward side from the uppermost position of the primary heat exchange portion 4a, flows in the row of the heat transfer tubes 6 on the leeward side to the lowermost position, and then flows out of the primary heat exchange portion 4a. The refrigerant in the refrigerant flow passage 280a that has flowed out of the primary heat exchange portion 4a flows into the primary heat exchange portion 4b of the heat exchanger block 10b from the lowermost position thereof, flows in the upward direction, and then flows out of the primary heat exchange portion 4b before the refrigerant reaches the uppermost position.

[0038] The refrigerant flowing through the refrigerant flow passage 280b flows in the row of the heat transfer tubes 6 on the windward side of the primary heat exchange portion 4a in a downward direction, flows out of the heat transfer tube 6 that is located in the lowermost

position, and then flows into the primary heat exchange portion 4b of the heat exchanger block 10b. The refrigerant in the refrigerant flow passage 280b that has flowed into the primary heat exchange portion 4b flows in the row of the heat transfer tubes 6 on the windward side of the primary heat exchange portion 4b to the uppermost position thereof, and flows into the row of the heat transfer tubes 6 on the leeward side from the uppermost position. The refrigerant flowing through the refrigerant flow passage 280b flows in the heat transfer tubes 6 on the leeward side of the primary heat exchange portion 4b in a downward direction, and flows out of the primary heat exchange portion 4b before the refrigerant reaches the lowermost position.

[0039] The refrigerant flow passage 280a and the refrigerant flow passage 280b are converged after the refrigerant flows out of the primary heat exchange portion 4b. The refrigerant converged at a converging portion 285 branches off from a bifurcation portion 286 again through a refrigerant flow passage 280c. The refrigerant flowing in a refrigerant flow passage 280d and a refrigerant flow passage 280e branched from the bifurcation portion 286 flows into the primary heat exchange portion 4c of the heat exchanger block 10c. The refrigerant flowing through the refrigerant flow passage 280d flows into the row of the heat transfer tubes 6 on the windward side of the primary heat exchange portion 4c, is transferred to the row on the leeward side of the primary heat exchange portion 4c from the uppermost position in the downward direction, and flows out of the primary heat exchange portion 4c at the lowermost position. The refrigerant that has flowed out of the primary heat exchange portion 4c flows into the lowermost position of the row of the heat transfer tubes 6 on the leeward side of the primary heat exchange portion 4d of the heat exchanger block 10d, flows in the upward direction, and flows out of the refrigerant outlet 283.

[0040] The refrigerant flowing through the refrigerant flow passage 280e flows in the row of the heat transfer tubes 6 on the windward side of the primary heat exchange portion 4c in a downward direction, flows out of the primary heat exchange portion 4c at the lowermost position. The refrigerant that has flowed out of the primary heat exchange portion 4c flows into the lowermost position of the row of the heat transfer tubes 6 on the windward side of the primary heat exchange portion 4d of the heat exchanger block 10d, flows in the upward direction, flows into the row of the heat transfer tubes 6 on the leeward side at the uppermost position, flows in the downward direction, and flows out of the refrigerant outlet 284.

[0041] As described above, the refrigerant flow passage 280 in the heat exchanger 201 includes the converging portion 285 that converges some or all of the branched refrigerant flow passages. Furthermore, the refrigerant flow passage 280 may include the bifurcation portion 286 at which the refrigerant flow passage branches off into the refrigerant flow passages equal in the number of refrigerant flow passages before being con-

verged at the converging portion 285. With such a configuration, the refrigerant flowing in the refrigerant flow passage 280a and the refrigerant flow passage 280b branched from the bifurcation portion 282 are converged at the converging portion 285 so that the refrigerant is mixed. In the case where the difference in heat load is generated for each portion of the refrigerant flow passage 280, the difference in the quality of the refrigerant flowing out of each of the refrigerant flow passage 280a and the refrigerant flow passage 280b can be reduced. Accordingly, the quality of the refrigerant branched in the heat exchanger 201 can be averaged, thereby being capable of reducing the variation in the temperature and humidity distribution of the air that passes through the heat exchanger 201. Furthermore, the risk that dew is condensed on the air-flow-direction control plate 3 arranged in the vicinity of the heat exchanger 201 can be reduced.

[0042] Note that the number of heat exchanger blocks 10 included in the heat exchanger 201 is not limited to four. Furthermore, the number of divided flow passages through which the air that has passed through the heat exchanger 1 passes may be appropriately changed in accordance with the number of heat exchanger blocks 10.

[0043] Fig. 5 is a cross sectional view illustrating a heat exchanger 201a in a modification example of the heat exchanger 201 according to Embodiment 2 of the present invention. As illustrated in Fig. 5, in the heat exchanger 201a, the auxiliary heat exchange portion 5 does not need to be included in the heat exchanger block 10. Regarding the primary heat exchange portion 4 and the auxiliary heat exchange portion 5 included in the heat exchanger block 10, the number of heat transfer tubes 6 in a row, the number of rows of the heat transfer tubes 6, and the tube diameter of the heat transfer tube 6 are not limited. In Embodiments 1 and 2, the number of refrigerant inlets 281 and refrigerant outlets 283, 284 are not limited to the number illustrated in Fig. 3 and Fig. 4. The number of flow passages branched at the bifurcation portion 286 after the refrigerant flow passage 280 is converged midway is not limited to the number equal to the number of refrigerant flow passages before being converged at the converging portion 285. Furthermore, a re-heat dehumidification valve may be provided on the downstream side of the converging portion 285 of the refrigerant flow passage 280.

<Effect of Embodiment 2>

[0044] (6) In the indoor unit 200 of an air-conditioning apparatus according to Embodiment 2, the refrigerant inlet 281 and the refrigerant outlets 283, 284 are provided in the heat exchanger block 10d closest to the rear air passage wall.

[0045] With such a configuration, a refrigerant inflow passage can be shortened between the heat exchanger 201 and a connection pipe between the outdoor unit and the indoor unit 100. This enables refrigerant pressure

loss to be reduced during the cooling operation, thereby improving the cooling performance of the air-conditioning apparatus. The amount of copper pipe used can be reduced by shortening the refrigerant inflow passage, thereby reducing the cost.

[0046] (7) In the indoor unit 200 of an air-conditioning apparatus according to Embodiment 2, the heat exchanger 201 includes the converging portion 285 at which at least one of the refrigerant flow passages 280a, 280b in which the branched refrigerant flows is converged to the refrigerant flow passage 280 continuous from the refrigerant inlet 281 to the refrigerant outlets 283, 284, and the bifurcation portion 286 that is provided on the downstream side of the flow of the refrigerant with respect to the converging portion 285, at which the refrigerant branches off into the refrigerant flow passages 280d, 280e again, the number of the refrigerant flow passages branched from a bifurcation portion 286 being greater than or equal to the number of refrigerant flow passages before being converged at the converging portion 285.

[0047] With such a configuration, the refrigerant flowing in the refrigerant flow passage 280a and the refrigerant flow passage 280b branched from the bifurcation portion 282 can be converged at the converging portion 285 so that the refrigerant can be mixed. Thus, in the case where the difference in heat load is generated for each portion of the refrigerant flow passage 280, the difference in the quality of the refrigerant flowing out of each of the refrigerant flow passage 280a and the refrigerant flow passage 280b can be reduced. Accordingly, the quality of the refrigerant branched in the heat exchanger 201 can be averaged, thereby being capable of reducing the variation in the temperature and humidity distribution of the air that passes through the heat exchanger 201. Furthermore, the risk that dew is condensed on the air-flow-direction control plate 3 arranged in the vicinity of the heat exchanger 201 can be reduced.

Reference Signs List

[0048]

1 heat exchanger 2 axial-flow fan 3 air-flow-direction control plate 4 primary heat exchange portion 4a primary heat exchange portion 4b primary heat exchange portion 4c primary heat exchange portion 4d primary heat exchange portion 5 auxiliary heat exchange portion 5a auxiliary heat exchange portion 5b auxiliary heat exchange portion 5c auxiliary heat exchange portion 5d auxiliary heat exchange portion 6 heat transfer tube 6a heat transfer tube 6b heat transfer tube 7 fin 10 heat exchanger block 10a heat exchanger block 10b heat exchanger block 10c heat exchanger block 10d heat exchanger block 20 drain pan 20a drain pan portion 20b drain pan portion 21a baffle plate 21b baffle plate 30 up-and-down airflow direction louver closer to front surface

31 rotation shaft 35 right-and-left airflow direction louver 40 up-and-down airflow direction louver closer to back surface 41 rotation shaft 50 casing 51 back panel 51a rear air passage wall 52 front panel 52a front air passage wall 55 air passage 56 front air passage 56a front air passage 56b central air passage 56c rear air passage 60 air inlet 70 air outlet 80 refrigerant flow passage 80a refrigerant flow passage 80b refrigerant flow passage 81 refrigerant inlet 82 bifurcation portion 83 refrigerant outlet 84 refrigerant outlet 100 indoor unit 200 indoor unit 201 heat exchanger 201a heat exchanger 280 refrigerant flow passage 280a refrigerant flow passage 280b refrigerant flow passage 280c refrigerant flow passage 280d refrigerant flow passage 280e refrigerant flow passage 281 refrigerant inlet 282 bifurcation portion 283 refrigerant outlet 284 refrigerant outlet 285 converging portion 286 bifurcation portion

Claims

1. An indoor unit (100, 200) for an air-conditioning apparatus comprising:

a casing (50);
an air inlet (60) provided on the casing (50);
an air outlet opening (70) in a bottom surface of the casing (50);
a heat exchanger (1) arranged in an air passage (55) extending from the air inlet (60) to the air outlet (70);
a fan (2) arranged in the air passage (55) between the air inlet (60) and the heat exchanger (1); and
an air-flow-direction control plate (3) provided in the air passage (55) between the heat exchanger (1) and the air outlet (70),
wherein the heat exchanger (1) comprises a plurality of heat exchanger blocks (10a, 10b, 10c, 10d) that are arranged in a front-and-rear direction of the casing (50), and the heat exchanger (1) includes:

a refrigerant inlet (81) through which refrigerant flows into the heat exchanger (1); and
a refrigerant outlet (83, 84) through which the refrigerant flows out of the heat exchanger (1),

the air-flow-direction control plate (3) is provided in a vicinity of one of the heat exchanger blocks (10a, 10b, 10c, 10d),
the refrigerant outlet (83, 84) is provided in a heat exchanger block (10d) other than the heat exchanger block (10a) that is provided in a vicinity of the air-flow-direction control plate (3),

and

the number of refrigerant outlets (83, 84) is larger than the number of refrigerant inlets (81), and wherein

the air passage (55) branches off into a plurality of divided air passages (56a, 56b, 56c) on a downstream side of the heat exchanger (1),
air that has passed through a first heat exchanger block (10a) without the refrigerant outlet (83, 84) among the plurality of heat exchanger blocks (10a, 10b, 10c, 10d) flows into a first divided air passage (56a) that is one of the plurality of divided air passages (56a, 56b, 56c), and
the air-flow-direction control plate (3) is disposed in the first divided air passage (56a).

2. The indoor unit (100, 200) for an air-conditioning apparatus of claim 1, wherein
the refrigerant inlet (81) and the refrigerant outlet (83, 84) are provided in a heat exchanger block (10d) closest to a rear air passage wall (51a).

3. The indoor unit (100, 200) for an air-conditioning apparatus of claim 1 or 2, wherein

the heat exchanger block (10a, 10b, 10c, 10d) includes:
an auxiliary heat exchange portion (5) and a primary heat exchange portion (4); and
the auxiliary heat exchange portion (5) is arranged so as to be overlapped with a windward side of the primary heat exchange portion (4) in the air passage (55),
the refrigerant inlet (81) is provided in the auxiliary heat exchange portion (5), and
the refrigerant outlet (83, 84) is provided in the primary heat exchange portion (4).

4. The indoor unit (100, 200) for an air-conditioning apparatus of any one of claims 1 to 3, further comprising:
a detector configured to detect a dry state of the refrigerant in the refrigerant outlet (83, 84).

5. An air-conditioning apparatus (100, 200), comprising:

an outdoor unit for an air-conditioning apparatus that includes a compressor configured to compress refrigerant; and
the indoor unit for an air-conditioning apparatus of any one of claims 1 to 4,
wherein a refrigeration cycle in which the refrigerant circulates between the outdoor unit and the indoor unit is constituted.

Patentansprüche

1. Inneneinheit (100, 200) für eine Klimaanlage, umfassend:

ein Gehäuse (50);
einen Lufteinlass (60), der an dem Gehäuse (50) vorgesehen ist;
eine Luftauslassöffnung (70) in einer unteren Oberfläche des Gehäuses (50);
einen Wärmetauscher (1), der in einem Luftdurchlass (55) angeordnet ist, der sich von dem Lufteinlass (60) zu dem Luftauslass (70) erstreckt;
einen Lüfter (2), der in dem Luftdurchlass (55) zwischen dem Lufteinlass (60) und dem Wärmetauscher (1) angeordnet ist; und
eine Luftströmungsrichtung-Steuerungsplatte (3), die in dem Luftdurchlass (55) zwischen dem Wärmetauscher (1) und dem Luftauslass (70) vorgesehen ist,
wobei der Wärmetauscher (1) eine Vielzahl von Wärmetauscherblöcken (10a, 10b, 10c, 10d) umfasst, die in einer vorder- und rückseitigen Richtung des Gehäuses (50) angeordnet sind, und der Wärmetauscher (1) aufweist:

einen Kältemittelinlass (81), durch den Kältemittel in den Wärmetauscher (1) strömt; und
einen Kältemittelauslass (83, 84), durch den das Kältemittel aus dem Wärmetauscher (1) herausströmt,

wobei die Luftströmungsrichtung-Steuerungsplatte (3) in einer Nähe eines der Wärmetauscherblöcke (10a, 10b, 10c, 10d) vorgesehen ist,
der Kältemittelauslass (83, 84) in einem anderen Wärmetauscherblock (10d) vorgesehen ist als dem Wärmetauscherblock (10a), der in einer Nähe der Luftströmungsrichtung-Steuerungsplatte (3) vorgesehen ist, und
die Anzahl von Kältemittelauslässen (83, 84) größer ist als die Anzahl von Kältemittelinlässen (81), und
wobei

der Luftdurchlass (55) auf einer stromabwärtigen Seite des Wärmetauschers (1) in eine Vielzahl von geteilten Luftdurchlässen (56a, 56b, 56c) abzweigt,
Luft, die durch einen ersten Wärmetauscherblock (10a) ohne den Kältemittelauslass (83, 84) aus der Vielzahl von Wärmetauscherblöcken (10a, 10b, 10c, 10d) hindurchgegangen ist, in einen ersten geteilten Luftdurchlass (56a) strömt, der einer

- der Vielzahl von geteilten Luftdurchlässen (56a, 56b, 56c) ist, und die Luftströmungsrichtung-Steuerungsplatte (3) im ersten geteilten Luftdurchlass (56a) angeordnet ist. 5
2. Inneneinheit (100, 200) für eine Klimaanlage nach Anspruch 1, wobei der Kältemiteleinlass (81) und der Kältemittelauslass (83, 84) in einem Wärmetauscherblock (10d) vorgesehen sind, der sich am nächsten zu einer rückseitigen Luftdurchlasswand (51a) befindet. 10
3. Inneneinheit (100, 200) für eine Klimaanlage nach Anspruch 1 oder 2, wobei 15
- der Wärmetauscherblock (10a, 10b, 10c, 10d) aufweist: einen zusätzlichen Wärmetauschabschnitt (5) und einen primären Wärmetauschabschnitt (4); 20 und der zusätzliche Wärmetauschabschnitt (5) so angeordnet ist, dass er sich mit einer Luvseite des primären Wärmetauschabschnitts (4) im Luftdurchlass (55) überlappt, 25 der Kältemiteleinlass (81) in dem zusätzlichen Wärmetauschabschnitt (5) vorgesehen ist, und der Kältemittelauslass (83, 84) in dem primären Wärmetauschabschnitt (4) vorgesehen ist. 30
4. Inneneinheit (100, 200) für eine Klimaanlage nach einem der Ansprüche 1 bis 3, ferner umfassend: einen Erfasser, der eingerichtet ist, einen trockenen Zustand des Kältemittels im Kältemittelauslass (83, 84) zu erfassen. 35
5. Klimaanlage (100, 200), umfassend:
- eine Außeneinheit für eine Klimaanlage, die einen Verdichter umfasst, der eingerichtet ist, Kältemittel zu verdichten; und 40 die Inneneinheit für eine Klimaanlage nach einem der Ansprüche 1 bis 4, wobei ein Kältekreislauf, in dem das Kältemittel zwischen der Außeneinheit und der Inneneinheit zirkuliert, gebildet ist. 45

Revendications

1. Unité intérieure (100, 200) d'un appareil de climatisation comprenant : 50
- une enveloppe (50) ; 55
- une entrée d'air (60) disposée sur l'enveloppe (50) ;
- une ouverture de sortie d'air (70) située dans une surface inférieure de l'enveloppe (50) ;

un échangeur de chaleur (1) agencé dans un passage d'air (55) qui s'étend à partir de l'entrée d'air (60) jusqu'à la sortie d'air (70) ; une soufflante (2) agencée dans le passage d'air (55) entre l'entrée d'air (60) et l'échangeur de chaleur (1) ; et une plaque de commande de la direction du flux d'air (3) disposée dans le passage d'air (55) entre l'échangeur de chaleur (1) et la sortie d'air (70), dans laquelle l'échangeur de chaleur (1) comprend une pluralité de blocs échangeurs de chaleur (10a, 10b, 10c, 10d) qui sont agencés dans la direction d'avant en arrière de l'enveloppe (50), et l'échangeur de chaleur (1) comprend :

une entrée de fluide frigorigène (81) à travers laquelle circule un fluide frigorigène dans l'échangeur de chaleur (1) ; et une sortie de fluide frigorigène (83, 84) à travers laquelle circule le fluide frigorigène hors de l'échangeur de chaleur (1) ;

la plaque de commande de la direction du flux d'air (3) est disposée à proximité de l'un des blocs échangeurs de chaleur (10a, 10b, 10c, 10d), la sortie de fluide frigorigène (83, 84) est disposée dans un bloc échangeur de chaleur (10d) différent du bloc échangeur de chaleur (10a) qui est disposé à proximité de la plaque de commande de la direction du flux d'air (3), et le nombre de sorties de fluide frigorigène (83, 84) est supérieur au nombre d'entrées de fluide frigorigène (81), et dans laquelle

le passage d'air (55) se ramifie en une pluralité de passages d'air divisés (56a, 56b, 56c) du côté aval de l'échangeur de chaleur (1), l'air qui est passé à travers un premier bloc échangeur de chaleur (10a) ne disposant pas de la sortie de fluide frigorigène (83, 84) parmi la pluralité de blocs échangeurs de chaleur (10a, 10b, 10c, 10d), circule dans un premier passage d'air divisé (56a) qui est l'un de la pluralité de passages d'air divisés (56a, 56b, 56c), et la plaque de commande de la direction du flux d'air (3) est disposée dans le premier passage d'air divisé (56a).

2. Unité intérieure (100, 200) d'un appareil de climatisation selon la revendication 1, dans laquelle l'entrée de fluide frigorigène (81) et la sortie de réfrigérant (83, 84) sont disposées dans le bloc échangeur de chaleur (10d) le plus proche d'une paroi de

passage d'air arrière (51a).

3. Unité intérieure (100, 200) d'un appareil de climatisation selon la revendication 1 ou 2, dans laquelle

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le bloc échangeur de chaleur (10a, 10b, 10c, 10d) comprend :

une partie échange de chaleur auxiliaire (5) et une partie échange de chaleur primaire (4) ; et

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la partie échange de chaleur auxiliaire (5) est agencée afin d'être chevauchée par un côté au vent de la partie échange de chaleur primaire (4) dans le passage d'air (55),

l'entrée de fluide frigorigène (81) est disposée dans la partie échange de chaleur auxiliaire (5), et

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la sortie de fluide frigorigène (83, 84) est disposée dans la partie échange de chaleur primaire (4).

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4. Unité intérieure (100, 200) d'un appareil de climatisation selon l'une quelconque des revendications 1 à 3, comprenant en outre :

un détecteur configuré pour détecter un état sec du fluide frigorigène dans la sortie de fluide frigorigène (83, 84).

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5. Appareil de climatisation (100, 200), comprenant :

une unité extérieure d'un appareil de climatisation qui comprend un compresseur configuré pour comprimer un fluide frigorigène ; et

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l'unité intérieure d'un appareil de climatisation selon l'une quelconque des revendications 1 à 4, dans laquelle est constitué un cycle de réfrigération dans lequel circule le fluide frigorigène entre l'unité extérieure et l'unité intérieure.

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FIG. 1

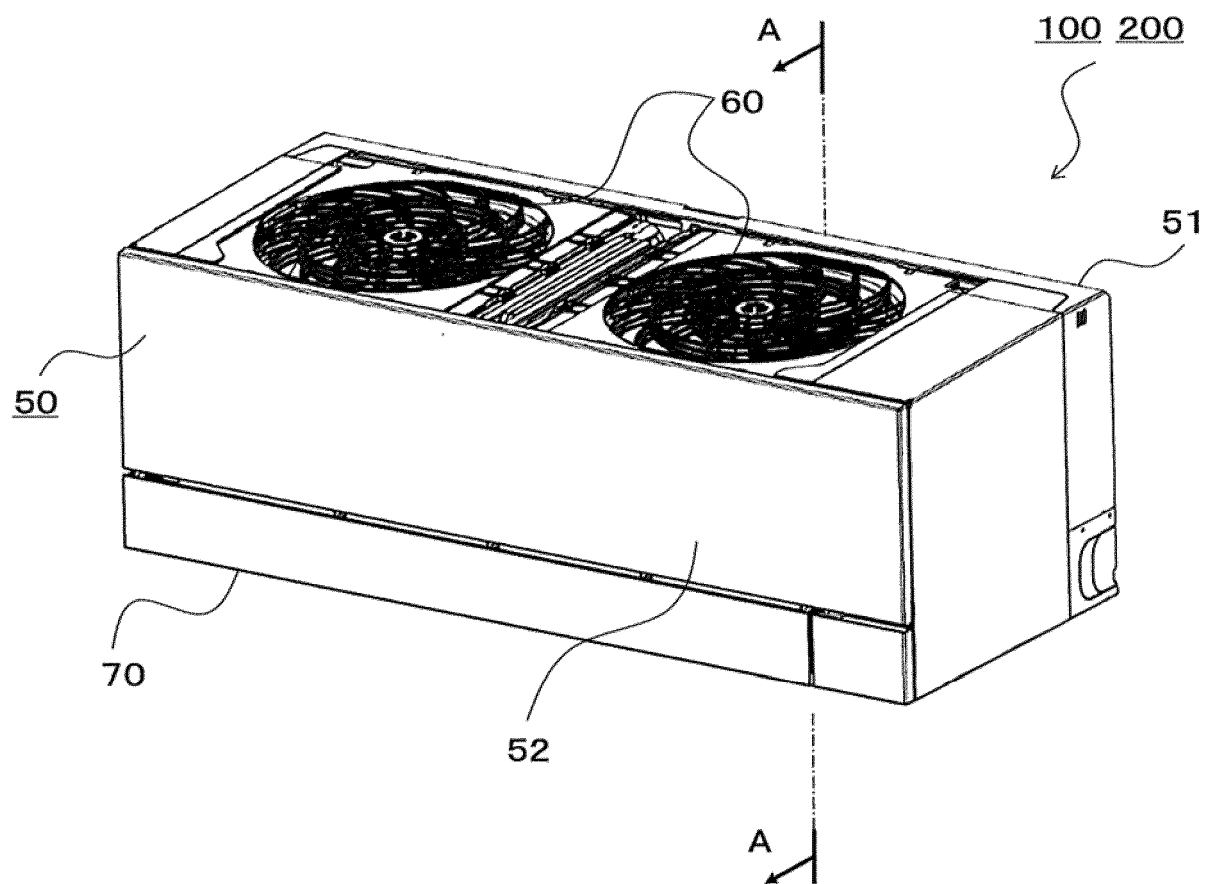


FIG. 2

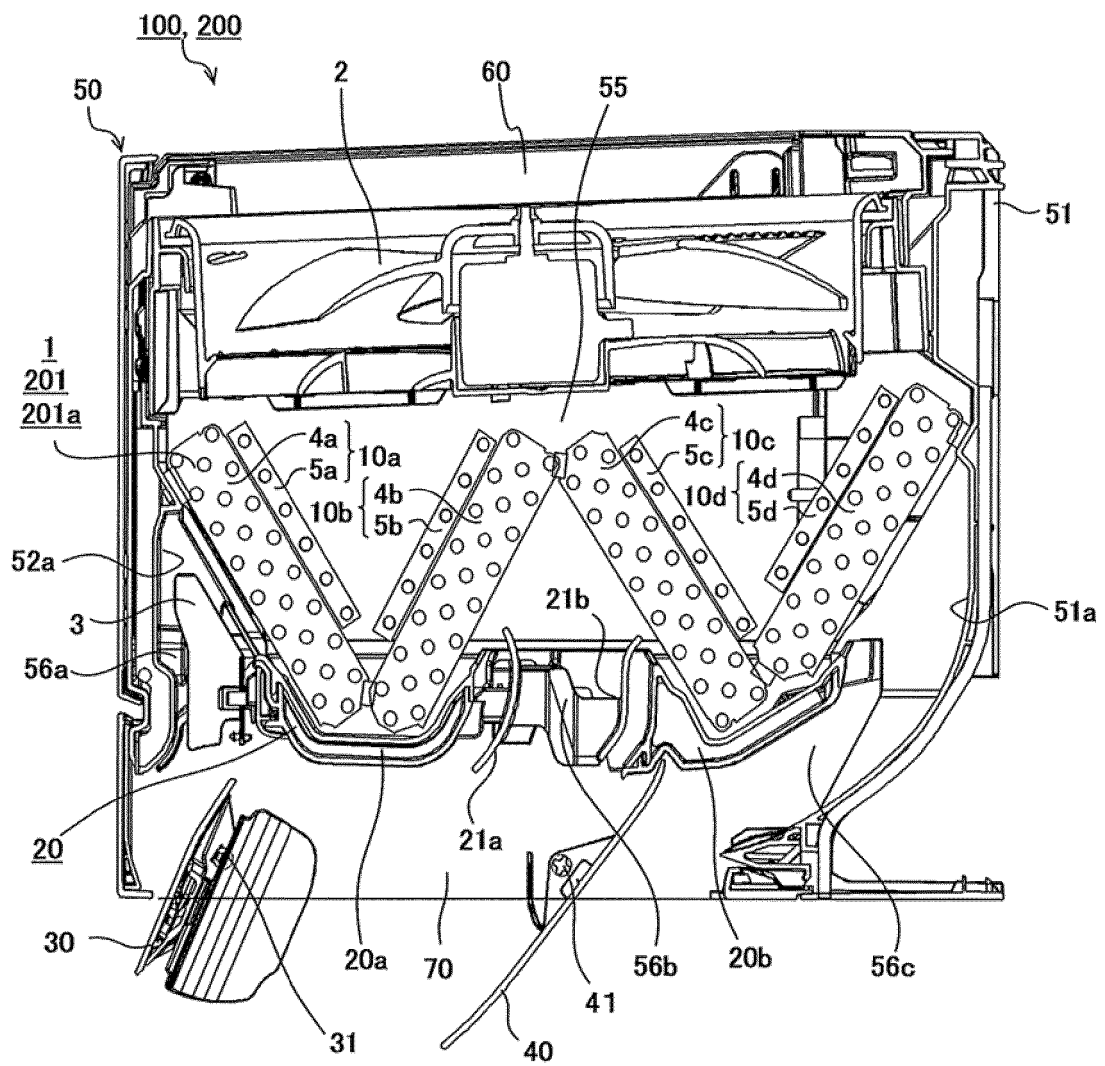


FIG. 3

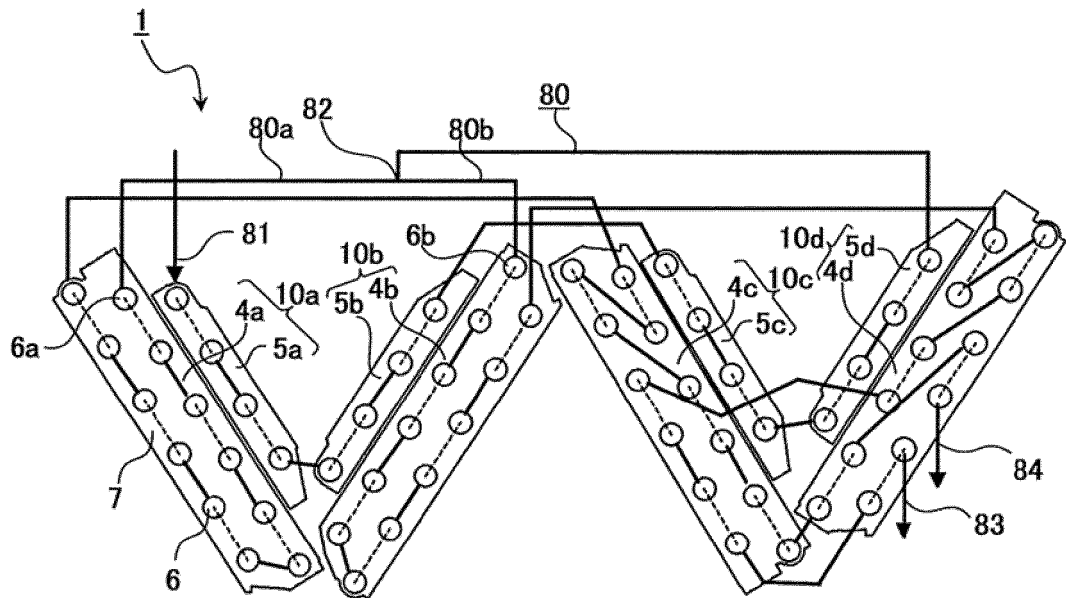


FIG. 4

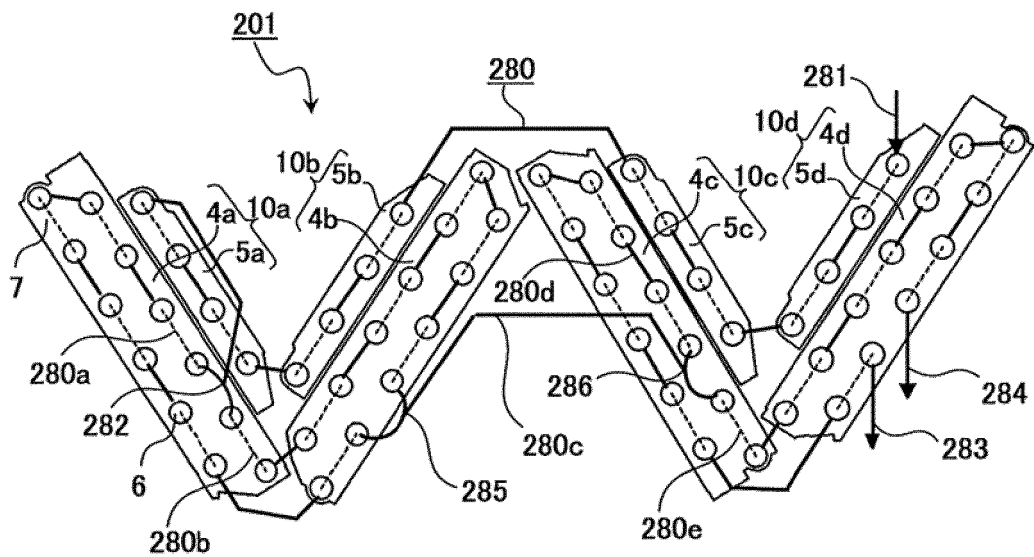
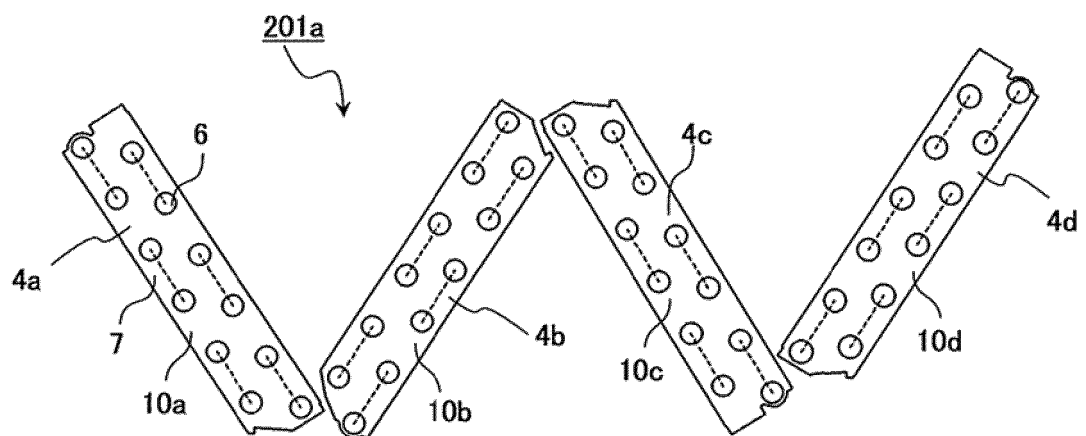


FIG. 5



REFERENCES CITED IN THE DESCRIPTION

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Patent documents cited in the description

- WO 2010089920 A [0009]
- WO 2016002015 A [0009]
- JP 5803898 B [0009]
- EP 2568226 A2 [0009]
- KR 20070053549 A [0009]
- JP 3045159 B [0009]