



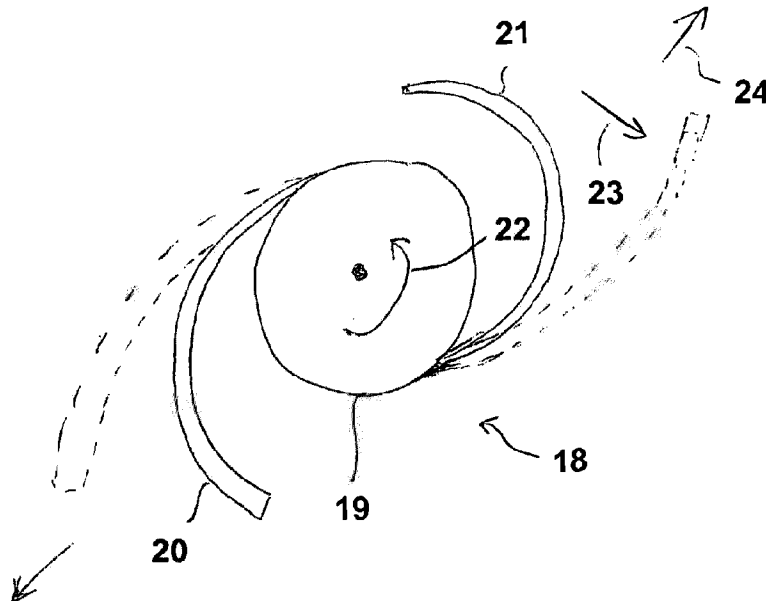
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(54) Title: FLUID PUMP WITH A ROTOR



(57) **Abrégé/Abstract:**

The invention relates to a fluid pump, in particular to a liquid pump having a rotor (18) with at least one rotor blade (20, 21) for conveying the fluid, the rotor being variable with respect to its diameter between a first, compressed state and a second expanded state. In order to produce simple compressibility and expandibility of the rotor of the pump, it is provided according to the invention that at least one rotor blade is deformable between a first state which it assumes in the compressed state of the rotor and a second state which it assumes in the expanded state of the rotor by means of a fluid counterpressure (23) during a rotation of the rotor during pump operation.

Abstract

The invention relates to a fluid pump, in particular to a liquid pump having a rotor (18) with at least one rotor blade (20, 21) for conveying the fluid, the rotor being variable with respect to its diameter between a first, compressed state and a second expanded state. In order to produce simple compressibility and expandability of the rotor of the pump, it is provided according to the invention that at least one rotor blade is deformable between a first state which it assumes in the compressed state of the rotor and a second state which it assumes in the expanded state of the rotor by means of a fluid counterpressure (23) during a rotation of the rotor during pump operation.

Fluid Pump with a Rotor

The invention resides in the field of fluid pumps and relates to a pump which is variable with respect to the rotor diameter thereof in order to be able to be guided for example through narrow openings, such as tubes, in particular blood vessels, and to be able to be operated in the expanded state after being guided through.

The invention can hence be provided, on the one hand, in the medical field, for example as blood pump for heart support, in a minimally invasive manner, however, on the other hand, use in agitators or as a propulsion element for ships is also conceivable.

The invention can exhibit particular advantages in the medical field as a result of possible miniaturisation.

After introducing the fluid pump through a large blood vessel into the ventricle and subsequently setting it in operation after expansion of the rotor, the pumping power of a heart can be assisted there considerably for example in humans or can be partially replaced. The therapeutic

advantage of such applications resides in an at least partial relief of the heart muscle.

Expandable fluid pumps of this type are known from the state of the art already. For example, a pump emerges from DE 10 059 714 C1 which can be pushed through a blood vessel together with the pump drive. The blood flows there through a cannula, the diameter of which can be expanded and compressed in order to change the flow ratios.

A blood pump, the rotor of which can be compressed and expanded radially, is known from WO 03/103745 A2, different constructions being proposed there in order to achieve the expandability. For example by means of different mutually displaceable parts of the catheter after introduction, compressing of the pump housing and radial widening, associated therewith, can be effected. On the other hand, by rotating a drive shaft relative to a wire located in the catheter, the possibility is disclosed of producing a helix structure of the wire, the wire carrying in addition a membrane which forms a rotor blade after assuming the helix structure.

In addition, a rotor structure having a plurality of blades which are rigid per se and articulated pivotably on a central part is known from the document, said blades being deployed during operation and hence producing a fluid pressure.

A pump is known from EP 0 768 900 B1, in which rotor blades are articulated on a shaft within a pump housing in such a manner that they can be folded against the shaft in the inoperative state and, during operation, can be deployed perpendicular to the shaft in order to convey the fluid.

From US 2006/0062672 A1, a rotor of a fluid pump is known with blades that are plially fixed to a hub and that are deployed by the fluid counterpressure generated by initial rotation of the rotor.

It is common to the known state of the art that rotor blades of a pump are pivoted either by means of a pivot mechanism for expansion or by fluid counterpressure during rotation or are formed by a mechanical device in the manner of a Bowden cable or the like only for expansion of the pump.

The object underlying the present invention with the background of the state of the art is to produce a fluid pump having a rotor which can be compressed with respect to the diameter thereof, which is built as simply as possible constructionally, which preferably comprises biocompatible materials like the pump housing surrounding it, the expansion and compression of which housing can be effected as simply as possible and which has the necessary reliability during operation.

The knowledge underlying the invention is that as simple a structure as possible of the fluid pump can be achieved by deformability of a rotor blade itself. The rotor of the fluid pump has, for this purpose, at least one rotor blade which is located in a first state as long as the rotor assumes a first, compressed state, the rotor blade assuming a second state during transition of the rotor into an expanded state by means of deformation.

The rotor blade is thereby transferred from the first state into the second state by the fluid counterpressure which occurs during rotation of the rotor during the pump operation.

A particular advantage of the invention resides in the fact that no actuation elements require to be provided for expansion of the rotor apart from the actual drive of the pump and in the fact that, due to the deformability of the rotor blade or blades per se, also no pivotable articulation of rotor blades to other parts of the pump requires to be provided.

The deformation of the blade is facilitated and at the same time delimited by providing a leading and a trailing side of the blade in the direction of movement during the conveying operation, wherein said sides have different configurations in the form of different material properties and/or constructional assembly at least along a part of the distance between the radially outer tip of the blade and the radially inner end of the blade.

The delimitation should thereby advantageously be, due to deformation, where a shape of the rotor which permits optimum conveying power is adopted. In other words, the deformability of the at least one rotor blade is advantageously delimited in such a manner that the deformation does not go beyond the shape in which the rotor produces the greatest possible fluid counterpressure.

When the fluid pump is being guided through a tube, for example a blood vessel, also no attempt is made by the rotor to expand without external influences. Such an attempt would not be desirable in medical use since the walls of the blood vessels through which the pump is guided should not be damaged. When applying through a tubular artificial access (valve), the described restoring forces would represent a particular difficulty since, as a result, high frictional forces would be

produced on the wall of the artificial tubes and significant forces would require to be produced to feed the fluid pump into the interior of the body.

As long as the pump is not being operated, i.e. is not rotated on the pump shaft, the rotor remains in the compressed state and can be fed through the blood vessel.

If the pump is put in operation in situ, then the rotor is actuated in the conveying direction and the rotor blade or blades are deformed by the fluid counterpressure and hence deployed, as a result of which the actual, extensive conveyance is set in motion. It is thereby advantageous if the deformation of the rotor blade/blades is elastic since, in many application cases, the fluid pump must be compressed again after application in order to be removed.

In this case, the rotor blade/blades assumes/assume their first state again, in which the rotor is compressed, after ceasing the pump operation and stopping the rotor.

Normally, the side of the rotor blade which is leading during operation (high pressure side) is predominantly subjected to tension whilst the trailing side (suction side) is subjected to a compressive stress. The interface between the leading and the trailing sides can thereby be imagined as where a neutral load is present in the pump operation. This interface must absorb corresponding transverse and shear stresses.

It can be provided for example that the leading side and the trailing side of the rotor blade are glued to each other in the region of the interface or are connected to each other by other joining techniques.

The properties of the rotor blade which are advantageous for the invention can be achieved for example in that the leading side of the at least one rotor blade comprises a first material and the trailing side comprises a second material which is different from the first. The two materials may be both different plastic, for example polymers with different properties, for example with different additives or one of them reinforced by fibers. It is also possible that one of the layers - preferably on the trailing side- comprises an elastomer and the other layer a polymer. The rotor blade also could be made of several thin layers of plastic material wherein each layer has different properties, e.g. a first layer with a low parameter, a second layer with a parameter higher than the first, a third layer with a parameter higher than the second layer etc. (the parameter may be any mechanical property or the like). If these layers are thin enough, the change of parameter over the thickness of the blade is (at least on a macroscopic scale) continuous. Such a plurality of layers may be manufactured by spraying and/or sputtering etc. of different materials for each layer.

It proves to be advantageous if the first material is more ductile than the second material.

The first material should thereby have a permanent elongation limit so that, during deformation of the rotor blade, a limit which is as precisely defined as possible is achieved during the pump operation and a defined shape of the rotor blade is set during operation. Such a permanent elongation limit is provided for example by a non-linear range of the coefficients of elasticity of the material so that the force required for elongation increases superproportionally from a specific permanent elongation limit and the shape is stabilised as a result. This property can be intrinsic to the first material but it can be assisted or essentially produced in that stretch-resistant fibres are embedded in the first material, said fibres being substantially more stretch-resistant than the first material itself and being present unstretched in the first

state of the rotor blade and in stretched form in the second state in the first material. Such fibres can be formed for example by high-strength plastic materials or by glass or by carbon fibres.

The second material on the trailing side of the rotor blades can be incompressible or be deformable only up to a specific compressibility limit. The deformability is advantageously elastic. The compression limit can be formed for example by a non-linearity of the compression coefficients in that the force required for the compression rises superproportionally from a specific compression degree.

It may also be advantageous if a first layer of material on the leading side and a second layer of material on the trailing side are provided wherein the second layer comprises trenches that allow for compression of said second layer up to the extent that the trenches are closed.

The trenches may be tangential to a circumferential direction of the rotor in order to allow for a bending of the rotor blade(s) along their radial length.

It can also be provided advantageously that the at least one rotor blade has, on the trailing side, shaped elements which are at a spacing from each other in the first state and abut against each other in the second state.

These shaped elements can be separated from each other in the first state by slots or also be embedded in a compressible material. At any rate they delimit further deformability of the rotor blade in that they abut against each other in the second state.

A further advantageous embodiment of the invention provides that at least one stop element is mounted on one side of the at least one rotor blade, said stop element penetrating the interface between the leading

side and the trailing side and being moveable in a limited manner in a recess on the other side of the rotor blade.

The stop element is advantageously produced from a material which is virtually as incompressible or precisely as incompressible as the material which the trailing side of the rotor blade comprises in order to achieve a defined stop position. The stop element can comprise for example a metal or a hard plastic material.

The invention relates, apart from to a fluid pump, in addition to a method for operating a fluid pump of the described form, the pump being started by rotation of the rotor in the operating direction and the rotor being expanded by the fluid counterpressure.

It can also be provided in addition that, in order to reduce the rotor diameter, the rotor is actuated in the direction opposite to the operating direction.

It is hence made possible by the invention that, when the pump is guided through an opening, in particular a blood vessel, the rotor is actuated in the direction opposite to the operating direction and hence is compressed.

The invention may also comprise that at least one rotor blade comprises at least one winglet for optimizing the fluidic conditions (see Figs. 13/14 in which "W" denotes a winglet and "B" denotes a blade). Fig. 15 shows an alternative embodiment with a winglet W' which is only on the leading side of the blade B'.

It may be advantageous that the at least one winglet protrudes from the leading side and/or from the trailing side of the blade.

The fluidic conditions between the rotor and the inner wall of a pump housing are best optimized by that at least one winglet is positioned at the tip of the blade. This winglet may also provide a bearing for the rotor insofar as it is gliding at the inner wall of the pump housing.

However, winglets may also be provided between the tip and the radially inner end of a blade where they can influence the fluid flow.

The winglets may be fixed pivotable with regard to the blade and maybe easily pivoted into their operating position by fluid pressure that is generated when the rotor is rotating (see Figs. 14 and 15).

The current invention also refers to a method for making a fluid pump as it has been described above.

The invention is illustrated subsequently in a drawing with reference to an embodiment and is explained subsequently.

There are thereby shown:

Fig. 1 schematically, the application of a fluid pump in a heart for conveying blood,

Fig. 2 schematically, a pump head in longitudinal section with radial inflow,

Fig. 2a schematically, a pump head in longitudinal section with axial inflow,

Fig. 3 schematically, a rotor with two rotor blades in a plan view,

Fig. 4 a rotor in a lateral view,

Fig. 5 a section through a part of a rotor blade,

- Fig. 6 a section through a part of a rotor blade in a different embodiment,
- Fig. 7 a section through a part of a rotor blade,
- Fig. 8 a sectional enlargement of the detail described in Fig. 7 with VIII,
- Fig. 9 a section through a rotor blade in a further embodiment,
- Fig. 10 an embodiment of a rotor with a helical rotor blade which is supported by shaped elements,
- Fig. 11 a rotor, the helical blade of which is supported by a spiral winding,
- Fig. 12 a rotor, the helical rotor blade of which is supported by a connecting member guide,
- Fig. 13 a perspective view of a blade with a winglet,
- Fig. 14 a sectional view of the device of Fig. 13,
- Fig. 15 a sectional view of an alternative design of a blade/winglet.

Fig. 1 shows schematically in cross-section a heart 1, in which the head 3 of a fluid pump protrudes into a ventricle 2. The pump head 3 is disposed at the end of a cannula 4 and has a pump housing 5 which is rounded at the front.

The drive of the pump is effected via a drive shaft 6 which extends longitudinally through the cannula 4 and is connected externally to a motor 7.

The motor 7 can be actuated in both directions 8, 9, conveyance of fluid actually taking place merely in one direction of rotation.

The pump head 3 with the pump housing 5 is shown schematically in Fig. 2 in longitudinal section and also the drive shaft 6. The latter is mounted rotatably at the front end of the pump head 3 in a bearing block 10 by means of a bearing 11.

Fig. 2 shows the pump head in an expanded form, i.e. with enlarged radius relative to the representation of Fig. 1.

For introduction of the pump head 3 through a blood vessel 12 into the heart, the pump head 3 is compressed radially by making the shaft slack or by axial pressure on the shaft, i.e. is brought into the state of its lowest possible radial elongation.

If the pump head has arrived at the desired location, then the pump housing can be drawn together axially by applying a tension in the direction of the arrow 13 and consequently can be expanded radially, as indicated by the arrows 14, 15.

Compression and expansion of the housing by deformation of the housing is also conceivable, by means of using shape memory materials. The resilient behaviour of shape memory materials at specific temperatures is hereby exploited. Through the slots 16, 17 which extend in the axial direction of the shaft 6, fluid, i.e. in the present case blood, can pass through the pump housing 5 towards the rotor 18 of the pump and can be conveyed further through the latter, for example axially through the cannula 4. In Fig. 2, the inflow of the

rotor has a radial configuration. In Fig. 2a, an embodiment with axial inflow and outflow is represented schematically.

The rotor has a rotor blade carrier 19 and also rotor blades 20, 21, the rotor blades 20, 21 being folded out during pump operation, i.e. in the expanded state of the rotor.

The radius of the rotor during operation is coordinated to the internal diameter of the pump housing in the expanded state thereof.

If the pump head is intended to be removed from the heart 1, then the pump operation is ceased and the rotor blades 20, 21 abut against the rotor blade carrier 19 in order to reduce the radius of the rotor 18. This is advantageously assisted by rotation of the rotor 18 in the direction of rotation opposite to the pump operation.

If the shaft 16 is then displaced towards the pump head 3 in the manner of a Bowden cable, then the pump head again assumes its compressed form and can be removed through the blood vessel 12.

Fig. 3 shows in detail a plan view on the rotor 18 with the rotor blade carrier 19 and the rotor blades 20, 21, these being represented in a continuous shape in their first state, i.e. the compressed state of the rotor. The rotor blades can also abut even more closely against the rotor blade carrier 19 in the first state.

It is important that, when the pump operation and rotation of the rotor 18 starts, in the direction of rotation 22 required for the conveyance operation, a fluid counterpressure is produced in the direction of the arrow 23 towards the rotor blades and these are bent by widening the radius of the rotor 18. If the pump is designed as a radial pump, then the fluid is displaced and hence conveyed radially outwards in the direction of the arrow 24.

If the rotor blades 20, 21 are profiled in the axial direction, then the fluid can be conveyed also in the axial direction, as indicated in Fig. 4 by the arrows 25, 26.

If the rotor is operated in a direction of rotation opposite to the direction of rotation 22 required for the conveyance, then a fluid counterpressure is produced on the rotor blades 20, 21, said counterpressure being opposite to the direction 23 and leading to the rotor blades folding up against the rotor blade carrier 19 and to a corresponding reduction in the rotor diameter. In this state, the rotor can be removed with a correspondingly compressed pump housing 5 out of the heart through the bloodstream.

By choice of the direction of rotation and the speed of rotation, the diameter of the rotor can hence be specifically changed, on the one hand, and, on the other hand, the conveyance power of the pump can be adjusted as desired.

Fig. 5 shows, by way of example, a rotor blade 21 with one side 27 which is leading during the pump operation and also a trailing side 28, the rotor blade having, along an interface 29, different properties on both sides thereof. During operation, a fluid counterpressure acts on the rotor blade in the direction of the arrow 23 and deforms the latter in the second state in which the rotor is expanded. For this purpose, the leading side 27 must be able to be elongated to a specific degree and the corresponding first material layer 30 has membrane properties for this reason. This first material layer can involve for example rubber or an elastic plastic material which is elastically deformable up to a permanent elongation limit and resists further elongation thereafter as far as possible.

On the trailing side 21, the second material layer 31 comprises a compression-resistant material which is configured for example to be so hard that it is deformed only minimally when forces are acting during operation so that bending of the rotor blade is produced exclusively via the elongation of the first material layer 30.

However, a certain compressibility of the second material layer 31 can be provided.

Fig. 6 shows a further example for configuration of a rotor blade in which notches 32 are provided in the second material layer 31, which allow compression and bending of the trailing side until the notches 32 are closed and the various webs formed between the notches 32 abut against each other in a form fit. In this state, further bending of the rotor blade would be stopped.

The material of the first material layer 31 in this case can likewise be a hard plastic material from which parts are cut out or recessed in a casting or embossing process.

In this case also, the material of the first material layer 30 comprises a material which can be elongated to a limited extent.

In Fig. 7, a rotor blade is represented in cross-section, the detail VIII in Fig. 8 being shown in more detail. The detail VIII thereby shows the compression-resistant second material layer 31a which, for its part, has a multilayer construction in the manner of a sandwich structure, the latter comprising tension- and/or compression-resistant external layers 33, 34, 35, 36 and also a volume layer 37. The external layers 35, 36 can be reinforced for example with a woven material.

A very compression-resistant layer is hence formed on the trailing side so that the deformability of the rotor blade is determined essentially by the ability of the leading side 27 to elongate.

In Fig. 9, a variant is represented in which a stop element 38 is mounted in the first layer 30, for example by means of a countersunk screw 39, the stop element 38 protruding into an opening 40 of the second layer 31.

If the rotor blade 21 is deformed, then the opening 40 in the second material layer 31 will tend to be reduced and displaced until the edges of the opening 40 abut against the stop element 38. The stop element comprises a hard material just like the second material layer 31 so that, after abutment, no further compression is possible on the trailing side and the paddle blade is reinforced against further deformation.

Fig. 10 shows a helical rotor blade in which a series of shaped elements 41, 42 on the trailing side of the blade are connected to each other, for example glued, or applied with a different joining method. In the compressed state of the rotor, a spacing exists between the shaped elements respectively. During operation of the pump and after deploying the blade, the shaped elements abut against each other and are reinforced as a continuous web which supports the flat parts of the blade acting as membrane and prevents further deformation. A plurality of such rows of shaped elements can be disposed along the drive shaft 6 axially and offset azimuthally.

A similar construction is shown in Fig. 11 where the web, for strengthening the rotor blade, is formed by a winding comprising coils, for example comprising a plastic material, a spring wire or a hose. The individual coils respectively form one shaped element and are connected individually to the membrane-like surface of the rotor blade by gluing. During compression of the rotor, the gussets between the windings and

open close these during deployment of the blade. In order to stabilise the winding, a continuous core is provided within the latter, said core being able to be flexible.

Fig. 12 shows the support of the rotor blade by a solid rail/connecting member 45 in which a stop element is moveable in a limited fashion. The stop element is connected to the rotor blade.

The rail/connecting member 45 can be configured, relative to the forces and moments which act as expected, as bend-resistant and compression-resistant component. As a result of the bending, small additional restoring forces are produced in this embodiment. Because of the low material thickness, regarded in absolute terms, few restoring forces are produced.

In Fig. 12, the stop element is located in the lower position. Bending up to the bent situation would require high acting forces for this position due to the small length between connecting member take-up on the shaft 6 and position of the guide pin in the rail/connecting member 45.

The mentioned and described constructions of rotor blades are examples of how, by means of different configuration of the various sides of the rotor blades, a limited deformability during operation can be achieved by the fluid counterpressure.

During rotation of the rotor in a direction opposite to the operating direction, the deformation of the rotor blades is reversed and these abut against the rotor, assume a first state and hence define the compressed state of the rotor in which the latter can move easily through a narrow opening, for example a blood vessel or a tubular artificial access (valve).

Hence the invention allows, in a constructionally particularly simple manner, production of a rotor which can be varied in its diameter for

various applications, but particularly advantageously for the medical field.

What is claimed is:

1. A fluid pump having a rotor with at least one rotor blade for conveying fluid, the rotor being variable with respect to its diameter between a first, compressed state and a second expanded state, wherein the at least one rotor blade is deformable
5 between a first state which it assumes in the compressed state of the rotor and a second state which it assumes in the expanded state of the rotor by means of a fluid counterpressure during a rotation of the rotor during pump operation, wherein the at least one rotor blade has a leading and a trailing side in a direction of movement during the conveying operation, characterised in that said leading
10 side and said trailing side have different configurations in a form of different material properties and/or constructional assembly at least along a part of a distance between a radially outer tip of the at least one rotor blade and a radially inner end of the at least one rotor blade, wherein the at least one rotor blade has a plurality of shaped elements created from the blade trailing side, said shaped
15 elements being selectively separated from one another by notches in said blade trailing side in said first state.
2. A fluid pump according to claim 1, characterised in that the at least one rotor blade is elastically deformable by means of the fluid counterpressure.
3. A fluid pump according to claim 1 or 2, characterised in that the deformability
20 of the at least one rotor blade is delimited by material properties and/ or additional support elements on one side of the rotor blade.

4. A fluid pump according to claim 3 characterised in that the deformability of the at least one motor blade is delimited in such a manner that the deformation does not go beyond the shape in which the rotor produces the greatest possible fluid counterpressure.
- 5 5. A fluid pump according to claim 4, characterised in that the leading side of the at least one rotor blade comprises a first material and the trailing side comprises a second material which is different from the first material.
6. A fluid pump according to claim 5, characterised in that the first material is more ductile than the second material.
- 10 7. A fluid pump according to claim 6, characterised in that the first material has a permanent elongation limit.
8. A fluid pump according to claim 6 or 7, characterised in that fibres are embedded into the first material and are substantially more stretch-resistant than the first material.
- 15 9. A fluid pump according to claim 5, characterised in that the second material is incompressible.
10. A fluid pump according to claim 5, characterised in that the second material is elastically compressible up to a compression limit.

11. A fluid pump according to claim 5, characterised in that a blade comprises a first layer of material on the leading side and a second layer of material on the trailing side and that the second layer comprises said notches that allow for compression of said second layer up to an extent that the notches are closed.
- 5 12. A fluid pump according to claim 1, characterised in that the shaped elements are glued on one side of the rotor blade or are applied by another joining method.
- 10 13. A fluid pump having a rotor with at least one rotor blade for conveying fluid, the rotor being variable with respect to its diameter between a first compressed state and a second expanded state, wherein the at least one rotor blade is deformable between a first state which it assumes in the compressed state of the rotor and a second state which it assumes in the expanded state of the rotor by means of a fluid counterpressure during a rotation of the rotor during pump operation, wherein the at least one rotor blade has a leading and a trailing side in a direction of movement during the conveying operation, characterised in that said leading side and said trailing side have different configurations in a form of different material properties and/or constructional assembly at least along a part of a distance between a radially outer tip of the at least one rotor blade and a radially inner end of the at least one rotor blade, the at least one rotor blade in its constructional assembly comprises a plurality of shaped elements on the trailing side or is connected to a plurality of shaped elements which are at a spacing from each other in the first state and abut against each other in the second state characterised in that the plurality of shaped elements are formed by respectively one coil of a winding.
- 15
- 20

14. A fluid pump having a rotor with at least one rotor blade for conveying fluid, the rotor being variable with respect to its diameter between a first, compressed state and a second expanded state, wherein the at least one rotor blade is deformable between a first state which it assumes in the compressed state of the rotor and a second state which it assumes in the expanded state of the rotor by means of a fluid counterpressure during a rotation of the rotor during pump operation, wherein the at least one rotor blade has a leading and a trailing side in a direction of movement during the conveying operation, characterised in that said leading side and said trailing side have different configurations in a form of different material properties and/or constructional assembly at least along a part of a distance between a radially outer tip of the at least one rotor blade and a radially inner end of the at least one rotor blade, the at least one rotor blade in its constructional assembly comprises a plurality of shaped elements on the trailing side or is connected to a plurality of shaped elements which are at a spacing from each other in the first state and abut against each other in the second state characterised in that the plurality of shaped elements are embedded in a compressible material.
15. A fluid pump according to claim 1, characterised in that at least one stop element is mounted on one side of the at least one rotor blade which penetrates the interface between the leading side and the trailing side and is moveable in a limited manner on the other side of the rotor blade in a recess.
16. A fluid pump according to claim 1, characterised in that at least one rotor blade comprises at least one winglet.

17. A fluid pump according to claim 16, characterised in that the at least one winglet protrudes from the leading side of the blade.
18. A fluid pump according to claim 16 or 17, characterised in that the at least one winglet protrudes from the trailing side of the blade.
- 5 19. A fluid pump according to claim 16, 17 or 18 characterised in that at least one winglet is positioned at the tip of the blade.
20. A fluid pump according to claim 15, 16, 17 or 18 characterised in that at least one winglet is pivotable with regard to the blade.
21. A fluid pump according to claim 20, characterised in that at least one pivotable
10 winglet is pivoted into its operating position by fluid pressure that is generated when the rotor is rotating.
22. A fluid pump having a rotor with at least one rotor blade for conveying fluid, the rotor being variable with respect to its diameter between a first, compressed state and a second expanded state, wherein the at least one rotor blade is deformable
15 between a first state which it assumes in the compressed state of the rotor and a second state which it assumes in the expanded state of the rotor by means of a fluid counterpressure during a rotation of the rotor during pump operation, wherein the at least one rotor blade has a leading and a trailing side in a direction of movement during the conveying operation, characterised in that said leading
20 side and said trailing side have different configurations in a form of different material properties and/or constructional assembly at least along a part of a

distance between a radially outer tip of the at least one rotor blade and a radially inner end of the at least one rotor blade, at least one rotor blade has a plurality of shaped elements on the trailing side, said plurality of shaped elements abut against each other as a continuous web in the second state.

- 5 23. A method of making a fluid pump having a rotor with at least one rotor blade for conveying fluid, the rotor being variable with respect to its diameter between a first, compressed state and a second expanded state, wherein the at least one rotor blade is deformable between a first state which it assumes in the compressed state of the rotor and a second state which it assumes in the
- 10 expanded state of the rotor by means of a fluid counterpressure during a rotation of the rotor during pump operation, wherein the at least one rotor blade has a leading and a trailing side in a direction of movement during the conveying operation, characterised in that said leading side and said trailing side have different configurations in a form of different material properties and/or
- 15 constructional assembly at least along a part of a distance between a radially outer tip of the at least one rotor blade and a radially inner end of the at least one rotor blade, wherein the at least one rotor blade has a plurality of shaped elements created from the blade trailing side, said shaped elements being selectively separated from one another by notches in said blade trailing side in
- 20 said first state, the method comprising:
- making said sides in different configurations in the form of different material properties and/or constructional assembly at least along a part of the distance between the radially outer tip of the blade and the radially inner end of the blade.

24. The method of making a fluid pump according to claim 23, further comprising making the at least one rotor blade elastically deformable by means of the fluid counterpressure.
- 5 25. The method of making a fluid pump according to claim 23 or 24, further comprising delimiting the deformability of the at least one rotor blade by material properties and/or additional support elements on one side of the rotor blade.
- 10 26. The method of making a fluid pump according to claim 25 wherein delimiting the deformability of the at least one rotor blade is characterised in that the deformation does not go beyond the shape in which the rotor produces the greatest possible fluid counterpressure.
27. The method of making a fluid pump according to claim 26, further comprising making the leading side of the at least one rotor blade from a first material and the trailing side from a second material which is different from the first.
- 15 28. The method of making a fluid pump according to claim 27, characterised in that the first material is more ductile than the second material.
29. The method of making a fluid pump according to claim 28, characterised in that the first material has a permanent elongation limit.

30. The method of making a fluid pump according to claim 28 or 29, characterised in that fibres are embedded into the first material and are substantially more stretch-resistant than the first material.
- 5 31. The method of making a fluid pump according to claim 28, characterised in that the second material is incompressible.
32. The method of making a fluid pump according to any one of claims 27 to 31, characterised in that the second material is elastically compressible up to a compression limit.
- 10 33. The method of making a fluid pump according to claim 27, characterised in that a blade comprises a first layer of material on the leading side and a second layer of material on the trailing side and that the second layer comprises said notches that allow for compression of said second layer up to an extent that the notches are closed.
- 15 34. The method of making a fluid pump according to claim 23, characterised in that the at least one rotor blade in its constructional assembly comprises shaped elements on the trailing side or is connected to shaped elements which are at a spacing from each other in the first state and abut against each other in the second state.
- 20 35. The method of making a fluid pump according to claim 34, characterised in that the shaped elements are separated from each other by slots in the first state.

36. The method of making a fluid pump according to claim 34 or 35, characterised in that the shaped elements are glued on one side of the rotor blade or are applied by another joining method.
37. The method of making a fluid pump according to claim 36, characterized in that
5 the one side of the rotor blade is the trailing side.
38. The method of making a fluid pump according to claim 34, 35, 36 or 37, characterised in that the shaped elements are formed by respectively one coil of a winding.
39. The method of making a fluid pump according to claim 34, characterised in that
10 the shaped elements are embedded in a compressible material.
40. The method of making a fluid pump according to claim 23, characterised in that at least one stop element is mounted on one side of the at least one rotor blade which penetrates the interface between the leading side and the trailing side and is moveable in a limited manner on the other side of the rotor blade in a recess.
- 15 41. The method of making a fluid pump according to claim 23, characterised in that at least one rotor blade comprises at least one winglet.
42. The method of making a fluid pump according to claim 41, characterised in that the at least one winglet protrudes from the leading side of the blade.

43. The method of making a fluid pump according to claim 41 or 42, characterised in that the at least one winglet protrudes from the trailing side of the blade.
44. The method of making a fluid pump according to claim 41, 42 or 43 characterised in that at least one winglet is positioned at the tip of the blade.
- 5 45. The method of making a fluid pump according to claim 40, 41, 42 or 43 characterised in that at least one winglet is pivotable with regard to the blade.
46. The method of making a fluid pump according to claim 45, characterised in that at least one pivotable winglet is pivoted into its operating position by fluid pressure that is generated when the rotor is rotating.
- 10 47. The method of making a fluid pump according to any one of claims 23 to 46, further comprising using two layers of different materials fixed to one another.
48. The method of making a fluid pump according to claim 47, characterised in that the two layers are glued or soldered to one another.
49. The method of making a fluid pump according to any one of claims 23 to 46,
15 characterised in that for making the blade, shaped bodies are fixed to one side of a layer.
50. A fluid pump, comprising a rotor with at least one rotor blade for conveying fluid, the rotor being variable with respect to its diameter between a compressed

- state, a first expanded state, and a second expanded state, wherein the at least one rotor blade deforms as the rotor transitions between the compressed state, the first expanded state, and the second expanded state, and wherein the second expanded state is achieved by means of a fluid counterpressure resulting from conveyance of fluid during rotation of the rotor during conveying pump operation, wherein the at least one rotor blade has a leading side and a trailing side with respect to the direction of rotation during conveying pump operation, wherein the rotor is configured such that when the rotor is rotating during conveying pump operation, the fluid counterpressure is produced in a direction which is counter to the direction of rotation acting against the leading side of the at least one rotor blade and the at least one rotor blade is resultingly deformed so as to widen the diameter of the rotor, and wherein in the second expanded state the at least one rotor blade forms a substantially straight line between a radially inner end of the rotor blade and a radially outer end of the rotor blade.
- 5
- 10
- 15 51. The fluid pump of claim 50, wherein the at least one rotor blade is elastically deformable by means of the fluid counterpressure.
52. The fluid pump of claim 50, wherein the deformability of the at least one rotor blade is delimited by material properties and/or additional support elements on one side of the rotor blade.
- 20 53. The fluid pump of claim 50, wherein the deformability of the at least one rotor blade is delimited in such a manner that the deformation does not go beyond a shape in which the rotor produces greatest possible fluid counterpressure.

54. The fluid pump of claim 53, wherein the leading side of the at least one rotor blade comprises a first material and the trailing side comprises a second material which is different from the first material.
55. The fluid pump of claim 54, wherein the first material is more ductile than the
5 second material.
56. The fluid pump of claim 54, wherein the first material has a permanent elongation limit.
57. The fluid pump of claim 54, wherein fibers are embedded into the first material and wherein the fibers are substantially more stretch-resistant than the first
10 material.
58. The fluid pump of claim 54, wherein the second material is incompressible.
59. The fluid pump of claim 54, wherein the second material is elastically compressible up to a compression limit.
60. The fluid pump of claim 54, wherein a blade comprises a first layer of material
15 on the leading side and a second layer of material on the trailing side and that the second layer comprises trenches that allow for compression of said second layer up to the extent that the trenches are closed.

61. The fluid pump of claim 50, wherein the at least one rotor blade in its constructional assembly comprises shaped elements on the trailing side or is connected to the shaped elements which are at a spacing from each other in the first state and abut against each other in the second state.
- 5 62. The fluid pump of claim 61, wherein the shaped elements are separated from each other by slots in the first state.
63. The fluid pump of claim 61, wherein the shaped elements are affixed on one side of the at least one rotor blade.
64. The fluid pump of claim 63, wherein the shaped elements are affixed on the
10 trailing side of the at least one rotor blade.
65. The fluid pump of claim 61, wherein the shaped elements are formed by respectively one coil of a winding.
66. The fluid pump of claim 61, wherein the shaped elements are embedded in a compressible material.
- 15 67. The fluid pump of claim 50, wherein at least one stop element is mounted on a first side of the at least one rotor blade which penetrates an interface between the leading side and the trailing side and is moveable in a limited manner on a second side of the at least one rotor blade in a recess.

68. The fluid pump of claim 50, wherein the at least one rotor blade comprises at least one winglet.
69. The fluid pump of claim 68, wherein the at least one winglet protrudes from the leading side of the at least one rotor blade.
- 5 70. The fluid pump of claim 68, wherein the at least one winglet protrudes from the trailing side of the blade.
71. The fluid pump of claim 68, wherein the at least one winglet is positioned at a tip of the at least one rotor blade.
72. The fluid pump of claim 68, wherein the at least one winglet is pivotable with
10 regard to the at least one rotor blade.
73. The fluid pump of claim 72, wherein the at least one pivotable winglet is pivoted into its operating position by fluid pressure that is generated when the rotor is rotating.
74. The fluid pump of claim 50, wherein the pump is designed as a radial pump such
15 that a fluid is displaced and conveyed radially outwards.
75. The method of claim 74, wherein a front edge of the rotor is substantially straight when the rotor reaches its maximum diameter.

76. The fluid pump of claim 50, wherein said leading and trailing sides of the rotor have different configurations in a form of different material properties and/or constructional assembly at least along a part of a distance between a radially outer tip of the at least one rotor blade and a radially inner end of the at least one rotor blade.
- 5
77. The fluid pump of claim 50, wherein the fluid pump is a liquid pump.
78. The fluid pump of claim 50, further comprising a housing, wherein the housing is comprised of shape memory metal.
79. The fluid pump of claim 50, further comprising a housing, wherein the housing is comprised of biocompatible materials.
- 10
80. The fluid pump of claim 79, wherein the rotor is comprised of biocompatible materials.
81. A method of making a fluid pump, comprising the step of making a rotor blade, wherein the fluid pump comprises a rotor with at least one rotor blade for conveying fluid, the rotor being variable with respect to its diameter between a compressed state, a first expanded state, and a second expanded state, wherein the at least one rotor blade deforms as the rotor transitions between the compressed state, the first expanded state, and the second expanded state, and wherein the second expanded state is achieved by means of a fluid counterpressure resulting from conveyance of fluid during rotation of the rotor during conveying pump operation, wherein the at least one rotor blade has a
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- leading side and a trailing side with respect to the direction of rotation during conveying pump operation, wherein the rotor is configured such that when the rotor is rotating during conveying pump operation, the fluid counterpressure is produced in a direction which is counter to the direction of rotation acting against the leading side of the at least one rotor blade and the at least one rotor blade is resultingly deformed so as to widen the diameter of the rotor, and wherein in the second expanded state the at least one rotor blade forms a substantially straight line between a radially inner end of the rotor blade and a radially outer end of the rotor blade.
- 5
- 10 82. The method of claim 81, wherein the blade is comprised of two layers of different materials fixed to one another.
83. The method of making the fluid pump of claim 82, wherein the two layers are affixed to one another.
84. The method of making the fluid pump of claim 82, further comprising affixing shaped bodies to one side of a layer.
- 15
85. A fluid pump, comprising a rotor with at least one rotor blade for conveying fluid, the rotor being variable with respect to its diameter between a compressed state, and an expanded state, wherein the at least one rotor blade deforms as the rotor transitions between the compressed state, and the expanded state, wherein the at least one rotor blade has a leading side and a trailing side with respect to the direction of rotation during conveying pump operation, and wherein the rotor is configured such that when the rotor is rotating during conveying pump operation, the fluid counterpressure is produced counter to the direction of
- 20

rotation against the leading side of the at least one rotor blade, wherein in the expanded state the at least one rotor blade forms a substantially straight line between a radially inner end of the rotor blade and a radially outer end of the rotor blade and wherein the rotor comprises a first material, wherein the first material comprises fibers.

- 5
86. The fluid pump of claim 85 wherein the fibers are stretch-resistant.
87. The fluid pump of claim 86 wherein the stretch-resistant fibers are more stretch-resistant than the first material.
88. The fluid pump of claim 86 wherein the stretch-resistant fibers are selected from the group comprising: high-strength plastic fibers, glass fibers, and carbon fibers.
- 10
89. The fluid pump of claim 86 wherein the stretch-resistant fibers are unstretched in the compressed state of the rotor.
90. The fluid pump of claim 86 wherein the stretch-resistant fibers are stretched in the expanded state of the rotor.
- 15
91. The fluid pump of claim 85 wherein the leading side is comprised of the first material and the trailing side is comprised of a second material.
92. The fluid pump of claim 91 wherein the first material differs from the second material.

93. The fluid pump of claim 85 wherein the rotor is variable with respect to its diameter between a compressed state, a first expanded state, and a second expanded state.
94. The fluid pump of claim 93, wherein the second expanded state is achieved by means of a fluid counterpressure resulting from conveyance of fluid during rotation of the rotor during conveying pump operation.
95. A fluid pump, comprising a rotor with at least one rotor blade for conveying fluid, the rotor being variable with respect to its diameter between a compressed state, and an expanded state wherein the at least one rotor blade has a leading side and a trailing side, wherein in the expanded state the at least one rotor blade forms a substantially straight line between a radially inner end of the rotor blade and a radially outer end of the rotor blade ,and wherein the leading side is made from a first material and wherein the first material comprises fibers.
96. The fluid pump of claim 95, wherein the fibers are stretch-resistant.
97. The fluid pump of claim 96, wherein the stretch-resistant fibers are more stretch-resistant than the first material.
98. The fluid pump of claim 96, wherein the stretch-resistant fibers are selected from the group comprising: high-strength plastic fibers, glass fibers, and carbon fibers.
99. The fluid pump of claim 96, wherein the stretch-resistant fibers are unstretched in the compressed state of the rotor.

100. The fluid pump of claim 96, wherein the stretch-resistant fibers are stretched in the expanded state of the rotor.
101. The fluid pump of claim 95, wherein the leading side is comprised of the first material and the trailing side is comprised of a second material.
- 5 102. The fluid pump of claim 101, wherein the first material differs from the second material.
103. The fluid pump of claim 95, wherein the rotor is variable with respect to its diameter between a compressed state, a first expanded state, and a second expanded state.
- 10 104. The fluid pump of claim 103, wherein the second expanded state is achieved by means of a fluid counterpressure resulting from conveyance of fluid during rotation of the rotor during conveying pump operation.
105. The fluid pump of claim 104, wherein the stretch-resistant fibers are stretched in the second expanded state of the rotor.
- 15 106. A fluid pump, comprising a rotor with at least one rotor blade for conveying fluid, the rotor being variable with respect to its diameter between a compressed state, a first expanded state, and a second expanded state, wherein the at least one rotor blade deforms as the rotor transitions between the compressed state, the first expanded state, and the second expanded state, and wherein the second
20 expanded state is achieved by means of a fluid counterpressure resulting from

conveyance of fluid during rotation of the rotor during conveying pump operation, wherein the at least one rotor blade has a leading side and a trailing side with respect to the direction of rotation during conveying pump operation, and wherein the rotor is configured such that when the rotor is rotating during conveying pump operation, the fluid counterpressure is produced counter to the direction of rotation against the leading side of the at least one rotor blade and the at least one rotor blade is resultingly deformed so as to widen the diameter of the rotor, and wherein the leading side comprises a first material and the trailing side comprises a second material and wherein the first material differs from the second material.

107. The fluid pump of claim 106, wherein the first material comprises fibers.
108. The fluid pump of claim 107, wherein the fibers are stretch-resistant.
109. The fluid pump of claim 108, wherein the stretch-resistant fibers are more stretch-resistant than the first material.
- 15 110. The fluid pump of claim 108, wherein the stretch-resistant fibers are selected from the group comprising: high-strength plastic fibers, glass fibers, and carbon fibers.
111. The fluid pump of claim 108, wherein the stretch-resistant fibers are unstretched in the compressed state of the rotor.

112. The fluid pump of claim 108, wherein the stretch-resistant fibers are stretched in the second expanded state of the rotor.
113. A percutaneous pump, comprising: a rotor with a hub and a helical rotor blade for conveying fluid, the rotor having a compressed state, a first expanded state, 5 and a second expanded state, wherein the helical rotor blade spans at least 180 degrees and the helical rotor blade is connected to the hub along a line forming a helix on a surface of the hub, the helical rotor blade being configured to pivot about the line on the surface of the hub; wherein the rotor is configured to increase in diameter between the compressed state and the first expanded state; 10 wherein the rotor is configured to increase in diameter between the first expanded state and the second expanded state while the helical rotor blade conveys fluid in a first rotational direction, said increase in diameter from the first expanded state due at least in part to fluid counterpressure acting against a leading side of the helical rotor blade in a second direction counter to the first 15 rotational direction.
114. The percutaneous pump of claim 113, wherein the percutaneous pump is located at a distal end of a drive shaft, and the percutaneous pump is driven by an external motor connected to a proximal end of the drive shaft.
115. The percutaneous pump of claim 114, wherein the rotor includes multiple helical 20 rotor blades.
116. The percutaneous pump of claim 113, wherein the rotor is formed at least in part from stretch-resistant fibers that are unstretched in the compressed state and stretched in both the first expanded state and the second expanded state.

117. The percutaneous pump of claim 113, wherein the rotor has a trailing side opposite the leading side, and wherein the leading side is formed from a first material and the trailing side is formed from a second material, different from the first material.
- 5 118. The percutaneous pump of claim 117, wherein the first material has fibers embedded therein, wherein the fibers are more stretch-resistant than the first material.
119. The percutaneous pump of claim 117, wherein the first material is more ductile than the second material.
- 10 120. A method of operating a fluid pump having a rotor (18) with at least one rotor blade (20, 21) for conveying fluid, the rotor being variable with respect to its diameter between a first, compressed state and a second expanded state, wherein the at least one rotor blade (20, 21) is deformable between a first state which it assumes in the compressed state of the rotor and a second state which it assumes
15 in the expanded state of the rotor by means of a fluid counterpressure (23) during a rotation of the rotor during pump operation, wherein the at least one rotor blade (20, 21) has a leading (27) and a trailing side (28) in the direction of movement during the conveying operation, wherein said sides have different configurations in the form of different material properties and/or constructional assembly at
20 least along a part of the distance between the radially outer tip of the blade and the radially inner end of the blade, the method comprising starting the pump by rotation of the rotor (18) in the operating direction; wherein the rotor is expanded by the fluid counterpressure (23).

121. A method of operating a fluid pump having a rotor (18) with at least one rotor blade (20, 21) for conveying fluid, the rotor being variable with respect to its diameter between a first, compressed state and a second expanded state, wherein the at least one rotor blade (20, 21) is deformable between a first state which it assumes in the compressed state of the rotor and a second state which it assumes in the expanded state of the rotor by means of a fluid counterpressure (23) during a rotation of the rotor during pump operation, wherein the at least one rotor blade (20, 21) has a leading (27) and a trailing side (28) in the direction of movement during the conveying operation, wherein said sides have different configurations in the form of different material properties and/or constructional assembly at least along a part of the distance between the radially outer tip of the blade and the radially inner end of the blade, the method comprising: in order to reduce the rotor diameter, the latter is actuated in the direction opposite to the operating direction.
122. A method of operating a fluid pump having a rotor (18) with at least one rotor blade (20, 21) for conveying fluid, the rotor being variable with respect to its diameter between a first, compressed state and a second expanded state, wherein the at least one rotor blade (20, 21) is deformable between a first state which it assumes in the compressed state of the rotor and a second state which it assumes in the expanded state of the rotor by means of a fluid counterpressure (23) during a rotation of the rotor during pump operation, wherein the at least one rotor blade (20, 21) has a leading (27) and a trailing side (28) in the direction of movement during the conveying operation, wherein said sides have different configurations in the form of different material properties and/ or constructional assembly at least along a part of the distance between the radially outer tip of the blade and the radially inner end of the blade, the method comprising: actuating the rotor

(18) when the pump is guided through an opening, in particular a blood vessel or a tubular artificial access, in the direction opposite to the operating direction.

5 123. A method for conveying fluid with a rotor, the method comprising: rotating a rotor in a first rotational direction, wherein a diameter of a helical rotor blade of the rotor increases between a compressed state and a first expanded state, wherein the helical rotor blade spans at least 180 degrees and is connected to a hub along a line forming a helix on a surface of the hub, and wherein the rotor pivots about the line on the surface of the hub; and rotating the rotor in the first rotational direction such that fluid counterpressure acts against a leading side of the helical rotor blade in a second direction which is counter to the first rotational direction, to increase the diameter of the helical rotor blade of the rotor between a first expanded state and a second expanded state.

124. The method of claim 123, further comprising: drawing a pump housing axially through a vessel, to introduce a pump head with the rotor into a heart.

15 125. The method of claim 123, further comprising: rotating the rotor in a second rotational direction which is counter to the first rotational direction, to decrease the diameter of the rotor.

126. The method of claim 125, further comprising: radially compressing the rotor to a state of lowest possible radial elongation to remove the rotor through a vessel.

20 127. The method of claim 126, further comprising: removing a pump head with the rotor from a heart.

128. A method for operating a fluid pump, comprising the step of starting rotation of a rotor in an operating direction, wherein the rotor is expanded by the fluid counterpressure, and wherein the fluid pump comprises the rotor with at least one rotor blade for conveying fluid, the rotor being variable with respect to its diameter between a compressed state, a first expanded state, and a second expanded state, wherein the at least one rotor blade deforms as the rotor transitions between the compressed state, the first expanded state, and the second expanded state, and wherein the second expanded state is achieved by means of a fluid counterpressure resulting from conveyance of fluid during rotation of the rotor-during conveying pump operation, wherein the at least one rotor blade has a leading side and a trailing side with respect to the direction of rotation during conveying pump operation, and wherein the rotor is configured such that when the rotor is rotating during conveying pump operation, the fluid counterpressure is produced counter to the direction of rotation against the leading side of the at least one rotor blade and the at least one rotor blade is resultingly deformed so as to widen the diameter of the rotor.
129. The method of claim 128, wherein in order to reduce the rotor diameter, the rotor is actuated in the direction opposite to the operating direction.
130. The method of claim 128, wherein the rotor is actuated when the pump is guided through an opening, in particular a blood vessel or a tubular artificial access, in the direction opposite to the operating direction.
131. The method of claim 128, wherein, under operation the diameter of the rotor is widened until it reaches a maximum.

132. A percutaneous pump, comprising: a rotor with a hub and a rotor blade for conveying fluid, the rotor having a compressed state, a first expanded state, and a second expanded state; wherein the rotor is configured to increase in diameter between the compressed state and the first expanded state; wherein the rotor is
5 configured to increase in diameter between the first expanded state and the second expanded state while the rotor blade conveys fluid in a first rotational direction, said increase in diameter from the first expanded state due at least in part to fluid counterpressure acting against a leading side of the rotor blade in a second direction counter to the first rotational direction; wherein the rotor blade
10 is a compressible and expandable continuous helical rotor blade; and wherein, in the second expanded state, the rotor blade forms a substantially straight line between a radially inner end of the rotor blade and a radially outer end of the rotor blade along a length of the continuous helical rotor blade.
133. The percutaneous pump of claim 132, wherein the percutaneous pump is located
15 at a distal end of a drive shaft, and the percutaneous pump is driven by an external motor connected to a proximal end of the drive shaft.
134. The percutaneous pump of claim 132, wherein the rotor includes multiple rotor blades.
135. The percutaneous pump of claim 132, wherein the rotor is formed at least in part
20 from stretch-resistant fibers that are unstretched in the compressed state and stretched in both the first expanded state and the second expanded state.
136. The percutaneous pump of claim 132, wherein the rotor has a trailing side opposite the leading side, and wherein the leading side is formed from a first

material and the trailing side is formed from a second material, different from the first material.

- 5 137. The percutaneous blood pump of claim 136, wherein the first material has fibres embedded therein, wherein the fibres are more stretch-resistant than the first material.
- 10 138. A method for conveying fluid with a rotor, the method comprising: rotating a rotor in a first rotational direction, wherein a diameter of a rotor blade of the rotor increases between a compressed state and a first expanded state; and rotating the rotor in the first rotational direction such that fluid counterpressure acts against a leading side of the rotor blade in a second direction which is counter to the first rotational direction, to increase the diameter of the rotor blade of the rotor between a first expanded state and a second expanded state, wherein the rotor blade is a compressible and expandable continuous helical rotor blade; and wherein, in the second expanded state, the rotor blade forms a substantially straight line between a radially inner end of the rotor blade and a radially outer end of the rotor blade along a length of the continuous helical rotor blade.
- 15 139. The method of claim 138, further comprising: drawing a pump housing axially through a vessel, to introduce a pump head with the rotor into a heart.
- 20 140. The method of claim 138, further comprising: rotating the rotor in a second rotational direction which is counter to the first rotational direction, to decrease the diameter of the rotor.

141. The method of claim 140, further comprising: radially compressing the rotor to a state of lowest possible radial elongation to remove the rotor through a vessel.
142. The method of claim 141, further comprising: removing a pump head with the rotor from a heart.
- 5 143. A percutaneous pump, comprising: a rotor with a hub and a rotor blade for conveying fluid, the rotor having a compressed state, a first expanded state, and a second expanded state, wherein a blade root of the rotor blade is connected to the hub at a connection point on a surface of the hub; wherein the rotor is configured to increase in diameter between the compressed state and the first expanded state;
10 state; wherein the rotor blade is a compressible and expandable rotor blade, and the blade root of the rotor blade forms a continuous helix on the hub; and wherein the rotor is configured to increase in diameter between the first expanded state and the second expanded state while the rotor blade conveys the fluid in a first rotational direction, said increase in diameter from the first
15 expanded state due at least in part to fluid counterpressure acting against a leading side of the rotor blade in a second direction counter to the first rotational direction.
144. The percutaneous pump of claim 143, wherein the percutaneous pump is located at a distal end of a drive shaft, and the percutaneous pump is driven by an
20 external motor connected to a proximal end of the drive shaft.
145. The percutaneous pump of claim 144, wherein the rotor includes multiple rotor blades.

146. The percutaneous pump of claim 137, wherein the rotor is formed at least in part from stretch-resistant fibers that are unstretched in the compressed state and stretched in both the first expanded state and the second expanded state.
- 5 147. The percutaneous pump of claim 143, wherein the rotor has a trailing side opposite the leading side, and wherein the leading side is formed from a first material and the trailing side is formed from a second material, different from the first material.
148. The percutaneous pump of claim 147, wherein the first material is more ductile than the second material.
- 10 149. A method for conveying fluid with a rotor, comprising: rotating a rotor in a first rotational direction, wherein a diameter of a rotor blade of the rotor increases between a compressed state and a first expanded state, wherein a blade root of the rotor blade is connected to a hub at a surface of the hub, and wherein the blade root of the rotor blade forms a continuous helix on the surface of the hub;
15 and rotating the rotor in the first rotational direction such that fluid counterpressure acts against a leading side of the rotor blade in a second direction which is counter to the first rotational direction to increase the diameter of the rotor blade of the rotor between a first expanded state and a second expanded state.
- 20 150. The method of claim 149, further comprising: drawing a pump housing axially through a vessel, to introduce a pump head with the rotor into a heart.

151. The method of claim 149, further comprising: rotating the rotor in a second rotational direction which is counter to the first rotational direction, to decrease the diameter of the rotor.
152. The percutaneous pump of claim 132, wherein the rotor blade is directly
5 connected to the hub.
153. The percutaneous pump of claim 132, wherein the rotor blade is connected to the hub at a connection point on a surface of the hub.
154. The percutaneous pump of claim 153, wherein the rotor blade is configured to pivot about the connection point on the surface of the hub.
- 10 155. The percutaneous pump of claim 154, wherein in the compressed state the rotor blade is configured to pivot about the connection point toward the hub.
156. The method of claim 138, wherein the rotor blade is connected to the hub at a connection point on a surface of the hub.
157. The method of claim 156, wherein the rotor blade is configured to pivot about
15 the connection point on the surface of the hub.
158. The method of claim 157, wherein in the compressed state the rotor blade is configured to pivot about the connection point toward the hub.

159. A percutaneous pump, comprising:

an expandable and compressible housing having a first inner diameter when compressed and a second inner diameter when expanded;

5 a rotor disposed within the housing, the rotor having a compressed state, a first expanded state, and a second expanded state; the rotor comprising:

a hub; and

a helical rotor blade coupled to the hub and configured to convey fluid, wherein the helical rotor blade spans at least 180 degrees about the hub;

10 wherein the helical rotor blade is configured to increase in diameter between the compressed state and the first expanded state to a first expanded diameter in the first expanded state, and to further increase between the first expanded diameter in the first expanded state to a second expanded diameter in the second expanded state, said increase to the second expanded diameter due at least in part to fluid counterpressure on the helical rotor blade;

15 wherein the helical rotor blade is configured to increase in diameter from the compressed state to the first expanded state due at least in part to expansion of the housing from the first inner diameter to the second inner diameter; and

wherein the rotor is configured to expand from the compressed state to the first expanded state and to the second expanded state within the expanded housing, and

20 wherein a first difference between the second inner diameter of the housing and the first expanded diameter of the helical rotor blade is greater than a

second difference between the second inner diameter of the housing and the second expanded diameter of the helical rotor blade.

- 5 160. The percutaneous pump of claim 159, wherein the helical rotor blade comprises a leading side of the blade and a trailing side of the blade, wherein the fluid counterpressure acts against the leading side of the blade.
161. The percutaneous pump of claim 160, wherein the helical rotor blade comprises a winglet at a distal tip of the blade.
- 10 162. The percutaneous pump of claim 161, wherein the winglet is configured to abut an inner wall of the housing when the helical rotor blade is in the second expanded state.
163. The percutaneous pump of claim 162, wherein the winglet is configured to glide against the inner wall of the housing during operation of the rotor.
164. The percutaneous pump of claim 162, wherein the winglet is pivotable with regard to the blade.
- 15 165. The percutaneous pump of claim 165, wherein the winglet is configured to be pivoted into an operating position by fluid pressure during rotation of the rotor.
166. The percutaneous pump of claim 165, wherein the winglet is provided on the leading side of the blade.

167. The percutaneous pump of claim 165, wherein the winglet is provided on the trailing side of the blade.
168. The percutaneous pump of claim 165, wherein the winglet protrudes from both the leading side and the trailing side of the blade.
- 5 169. The percutaneous pump of claim 159, wherein the housing is rounded at distal end.
170. The percutaneous pump of claim 159, wherein the percutaneous pump is located at a distal end of a drive shaft, and the percutaneous pump is driven by an external motor connected to a proximal end of the drive shaft.
- 10 171. The percutaneous pump of claim 170, wherein the drive shaft is flexible and is configured to extend over an aortic arch.
172. The percutaneous pump of claim 170, wherein the housing comprises a shape memory material.
- 15 173. The percutaneous pump of claim 170, wherein the housing is configured to be radially expanded by application of a force in a proximal direction along a longitudinal axis of the housing.

174. The percutaneous pump of claim 173, wherein the force in the proximal direction is applied to the drive shaft relative to the housing to expand the housing to the second inner diameter.
- 5 175. The percutaneous pump of claim 159, wherein the housing further comprises slots configured to allow a fluid to flow into an interior of the housing.
176. The percutaneous pump of claim 159, wherein a diameter of the rotor in the compressed state is less than the first inner diameter of the housing when compressed.
- 10 177. The percutaneous pump of claim 159, wherein the rotor includes multiple helical rotor blades.
178. The percutaneous pump of claim 159, wherein the helical rotor blade is pivotable about a line on the surface of the hub.
179. The percutaneous pump of claim 159, wherein the housing surrounds the helical rotor blade in the compressed state and in the first expanded state.
- 15 180. The percutaneous pump of claim 159, wherein the housing is not longitudinally movable with respect to the helical rotor blade.
181. The percutaneous pump of claim 159, further comprising a sheath surrounding the housing and movable with respect to the housing.

182. The percutaneous pump of claim 159, wherein a distance from the distal end of the housing to a distal end of the helical rotor blade is the same when the housing is compressed as when the housing is expanded.

183. A percutaneous pump, comprising:

5 a rotor with a hub and a helical rotor blade for conveying fluid, the rotor having a compressed state and an expanded state;

wherein the helical rotor blade comprises a leading side and a trailing side with respect to a direction of rotation during pump operation, the leading side
10 comprising a first material configured to be elastically deformable and the trailing side comprising a second material configured to be compression-resistant;

wherein the rotor is configured to increase in diameter between the
15 compressed state and the expanded state, said increase in diameter from the compressed state to the expanded state due at least in part to fluid counterpressure acting against the leading side of the helical rotor blade; and

wherein the leading side is elongated when in the expanded state.

20 184. The percutaneous pump of claim 183, wherein the trailing side is minimally deformed when in the expanded state.

185. The percutaneous pump of claim 183, wherein the leading side is configured as a first face of the helical rotor blade extending from a blade root of the helical rotor blade to a blade tip of the helical rotor blade and the trailing side is

configured as a second face of the helical rotor blade extending from the blade root to the blade tip.

186. The percutaneous pump of claim 183, wherein the first material is comprised of a rubber or an elastic plastic material.
- 5 187. The percutaneous pump of claim 183, wherein the second material is comprised of a hard plastic material.
188. The percutaneous pump of claim 183, wherein the first material or the second material is comprised of a plurality of layers.
189. The percutaneous pump of claim 183, wherein the first material is more ductile
10 than the second material.
190. The percutaneous pump of claim 183, wherein the first material is configured to have a permanent elongation limit, wherein the permanent elongation limit is configured to be determined by a property of the first material.
191. The percutaneous pump of claim 183, wherein a deformability of the helical
15 rotor blade is configured to be determined by a property of the leading side related to elongation.
192. The percutaneous pump of claim 190, wherein the property is a non-linear range of coefficients of elasticity of the first material.

193. A percutaneous pump, comprising:

a rotor with a hub and a helical rotor blade for conveying fluid, the rotor having a

compressed state and an expanded state;

5 wherein the helical rotor blade comprises a leading side and a trailing side with respect to a direction of rotation during pump operation, the trailing side comprising a plurality of shape elements;

wherein the rotor is configured to increase in diameter between the compressed state and the expanded state, said increase in diameter from the compressed state to the expanded state due at least in part to fluid
10 counterpressure acting against the leading side of the helical rotor blade; and

wherein in the expanded state a first shape element of the plurality of shape elements abuts a second shape element of the plurality of shape elements.

194. The percutaneous pump of claim 190, wherein the leading side is configured as a
15 first face of the helical rotor blade extending from a blade root of the helical rotor blade to a blade tip of the helical rotor blade and the trailing side is configured as a second face of the helical rotor blade extending from the blade root to the blade tip.

195. The percutaneous pump of claim 194, wherein the plurality of shape elements
20 are disposed axially on the trailing side of the helical rotor blade from the blade root to the blade tip.

196. The percutaneous pump of claim 194, wherein the plurality of shape elements are joined to the second face of the trailing side of the helical rotor blade and configured to support the trailing side of the helical rotor blade to prevent deformation in the expanded state.
- 5 197. The percutaneous pump of claim 193, wherein the plurality of shape elements are configured to be connected to a surface of the helical rotor blade by glue.
198. The percutaneous pump of claim 193, wherein the plurality of shape elements are a winding comprised of a plurality of coils, where each shape element is configured to be a coil of the plurality of coils.
- 10 199. The percutaneous pump of claim 198, wherein the plurality of coils are comprised of a plastic material, a spring wire, or a hose.
200. The percutaneous pump of claim 198, wherein each coil of the plurality of coils is joined to a face of the trailing side of the helical rotor blade.
- 15 201. The percutaneous pump of claim 193, wherein the plurality of shape elements are embedded in a compressible material.
202. The percutaneous pump of claim 193, wherein in the compressed state the first shape element is offset from the second shape element.

203. The percutaneous pump of claim 202, wherein in the compressed state the first shape element is offset azimuthally from the second shape element.

204. The percutaneous pump of claim 203, wherein in the compressed state the first shape element is offset azimuthally from the second shape element by a slot in a
5 face of the trailing side of the helical rotor blade.

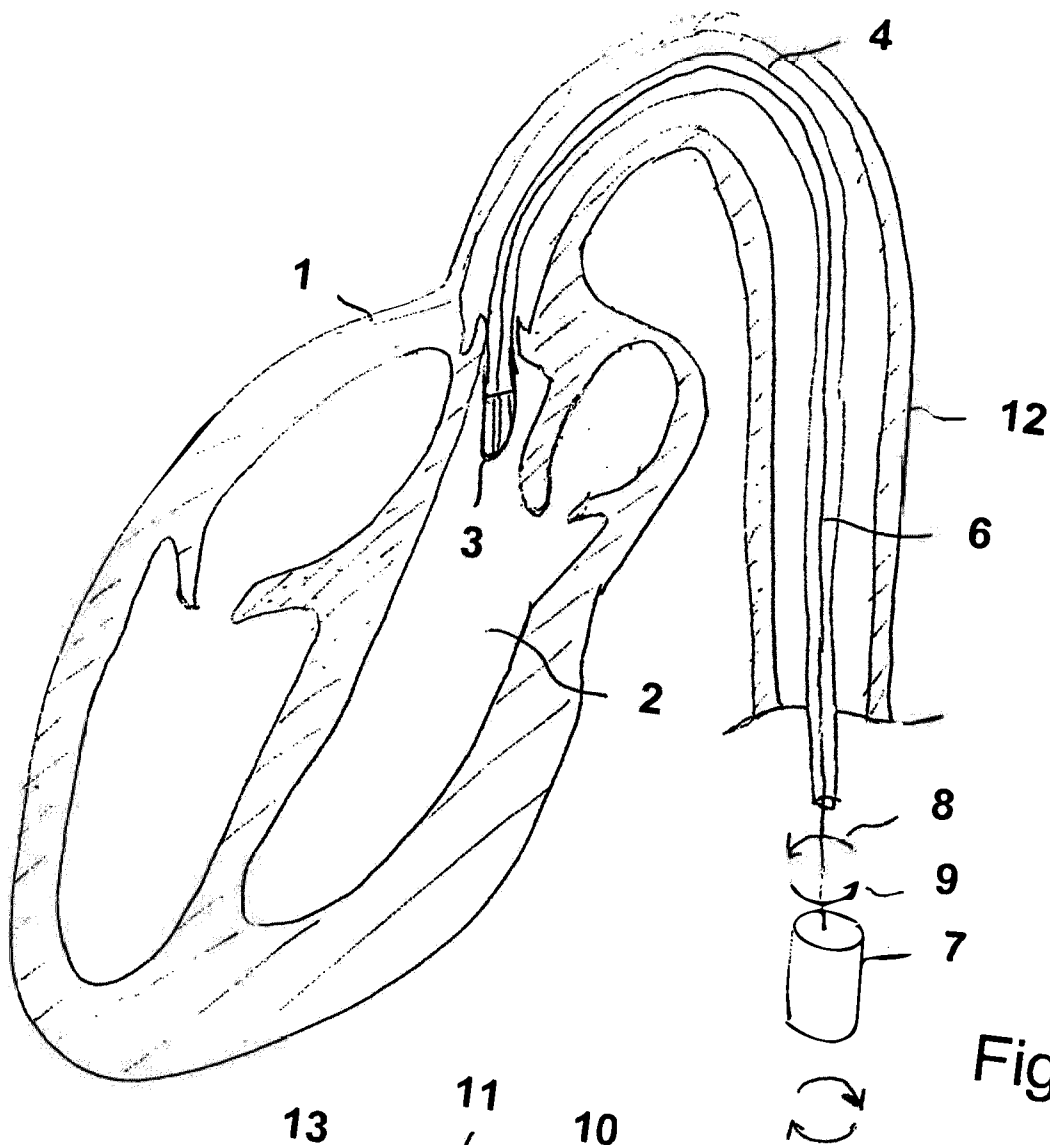


Fig. 1

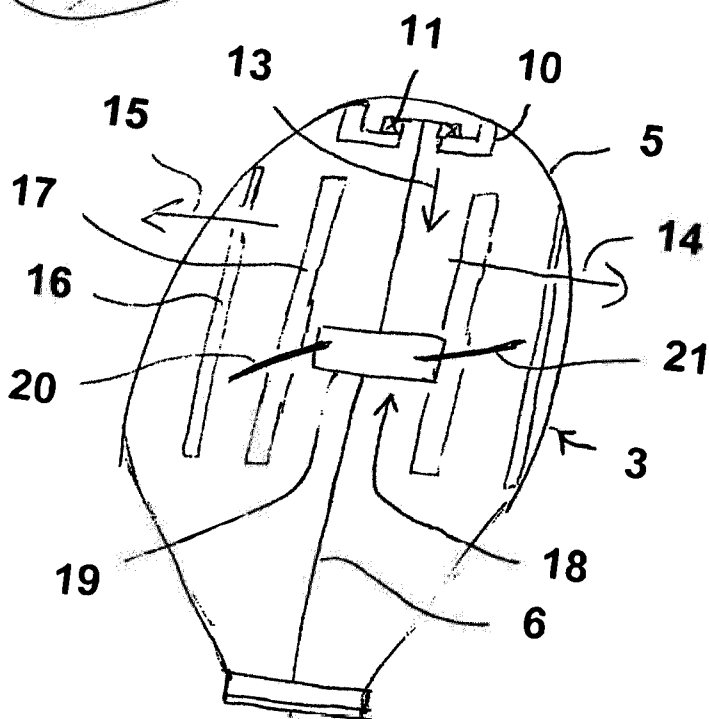


Fig. 2

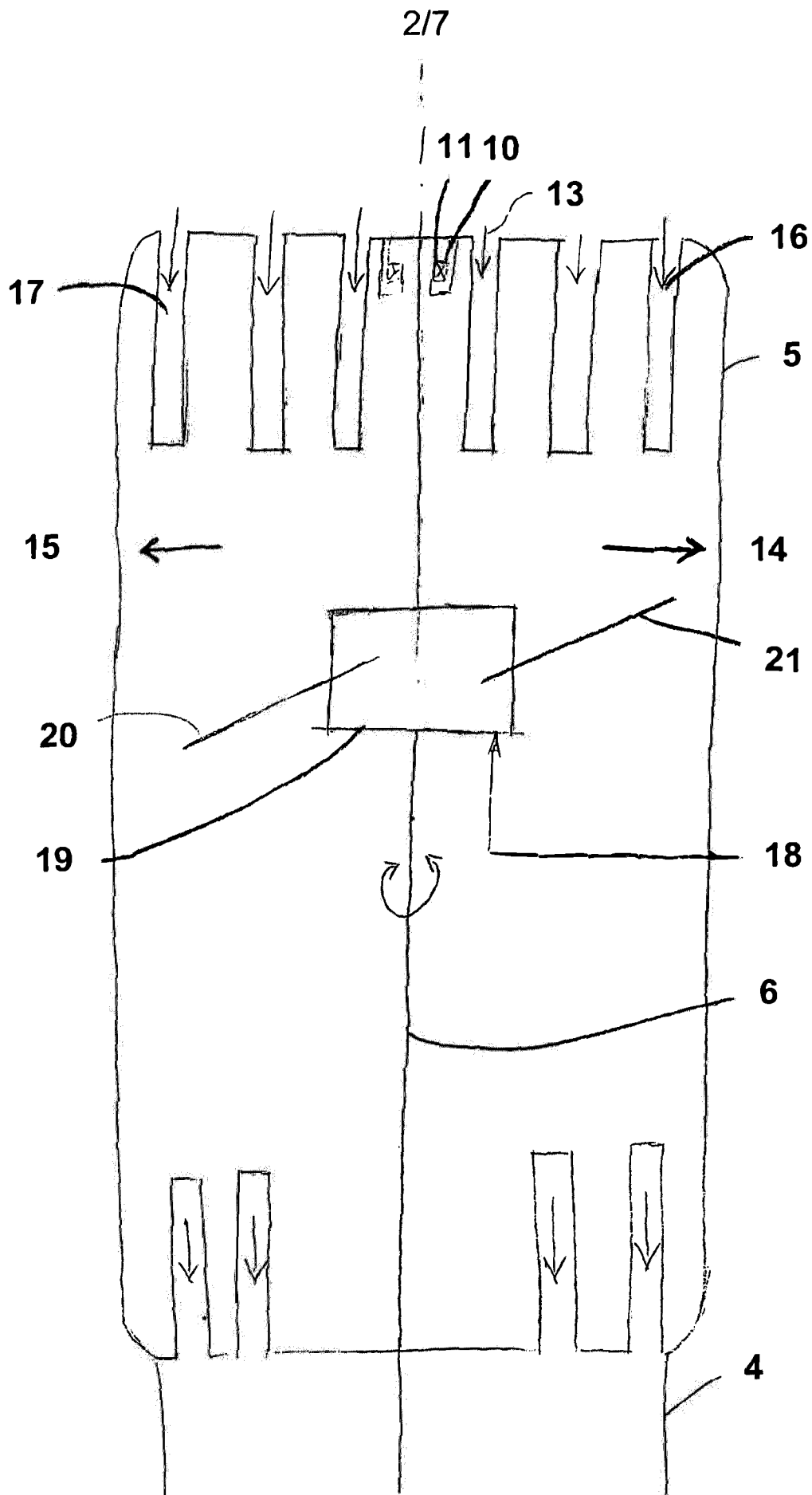


Fig. 2a

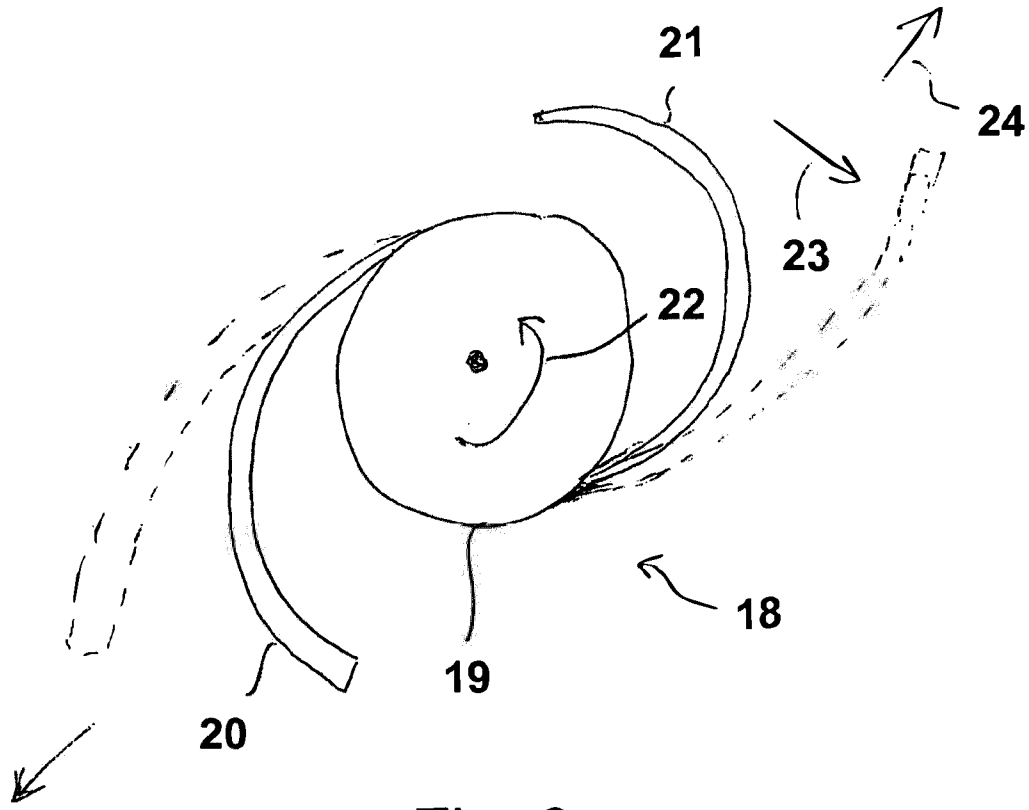


Fig. 3

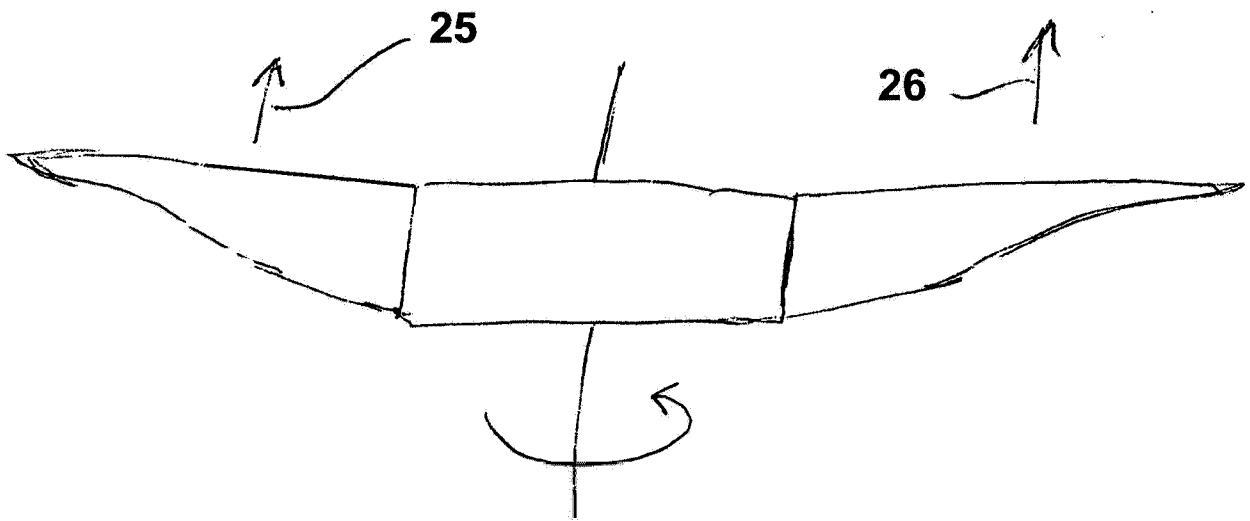


Fig. 4

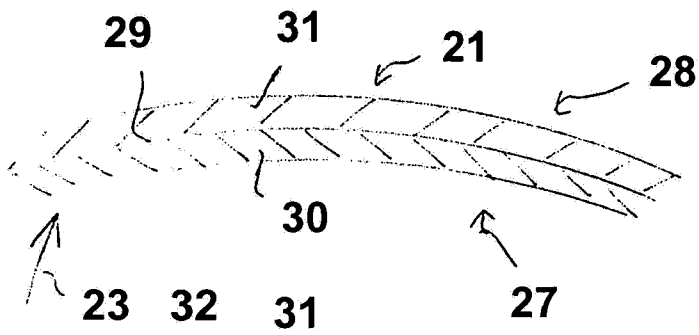


Fig. 5

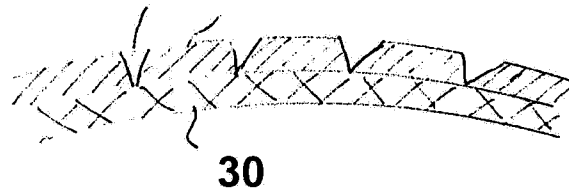


Fig. 6

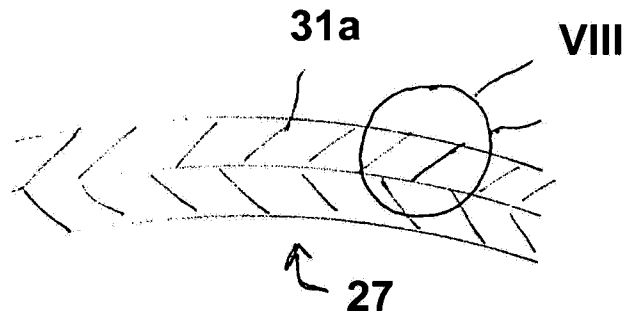


Fig. 7

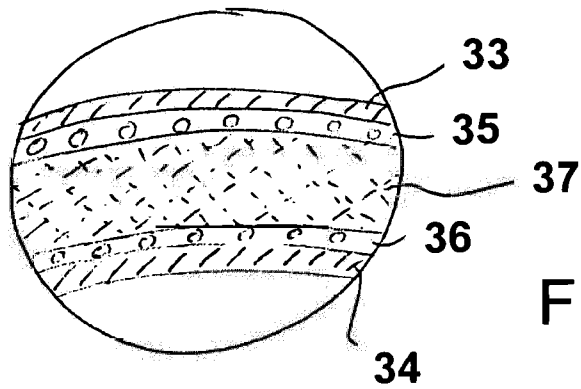


Fig. 8

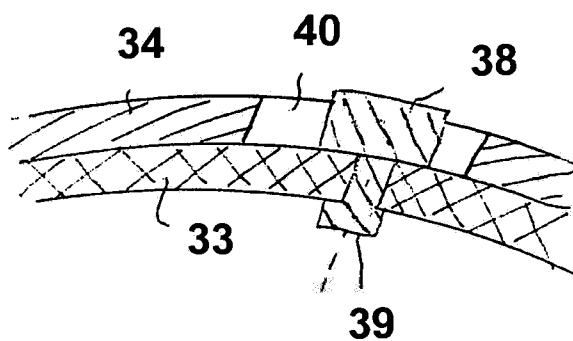


Fig. 9

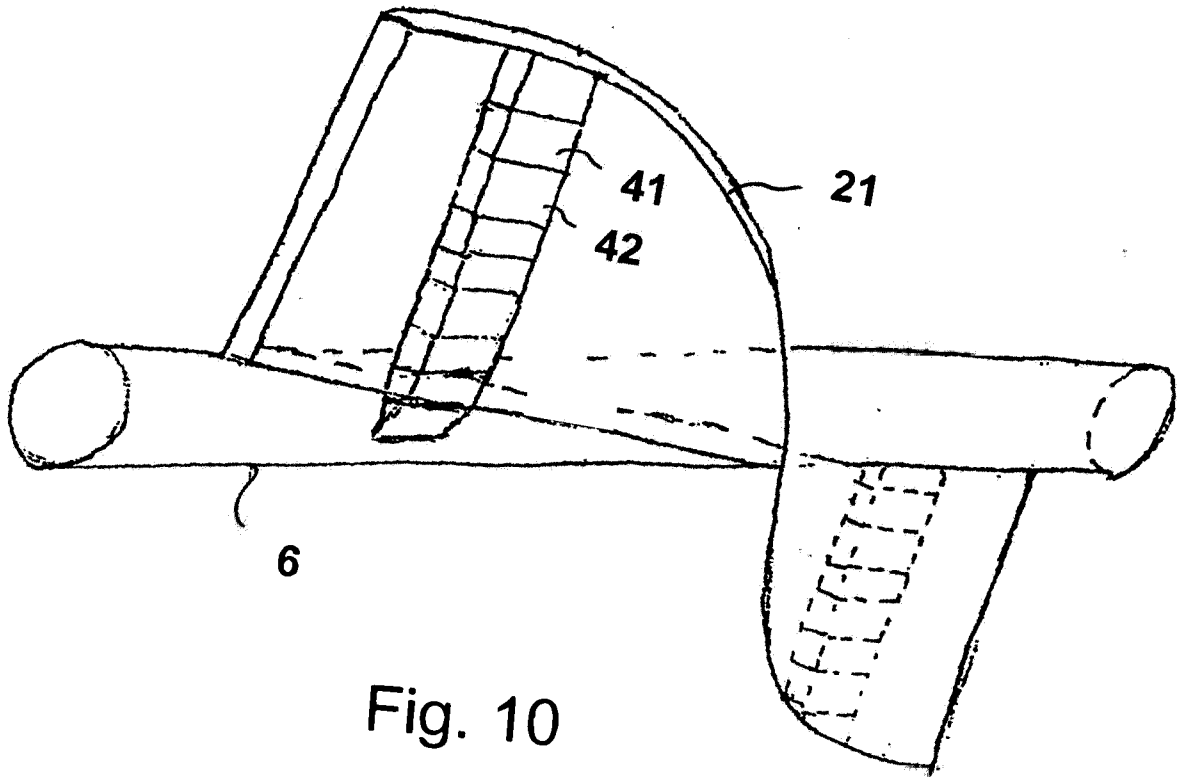


Fig. 10

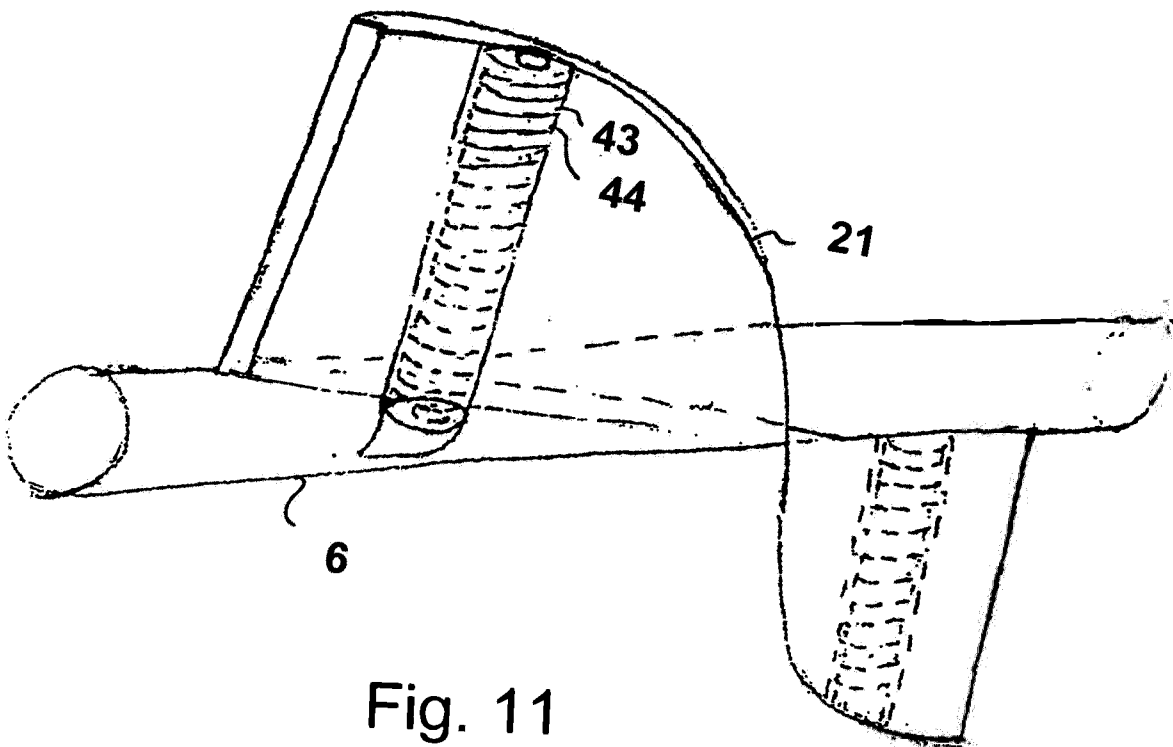


Fig. 11

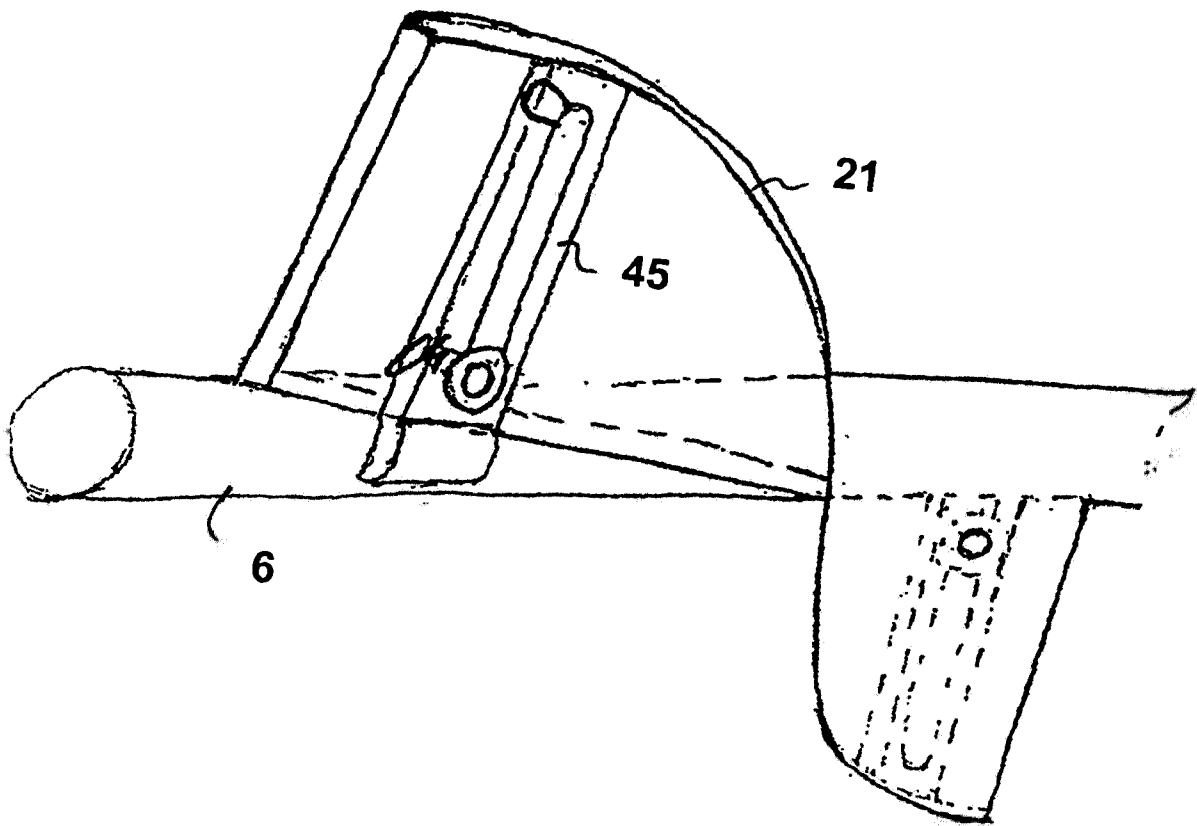


Fig. 12

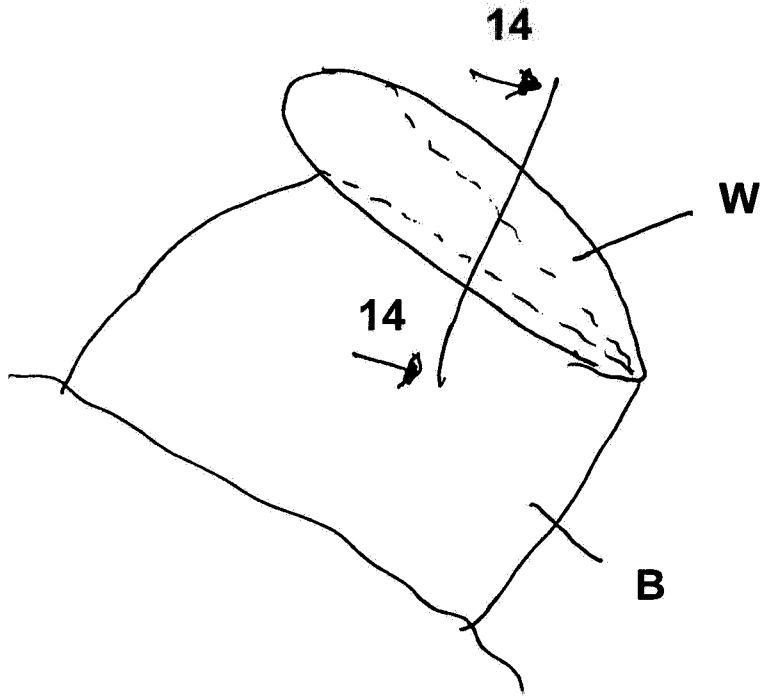


Fig. 13

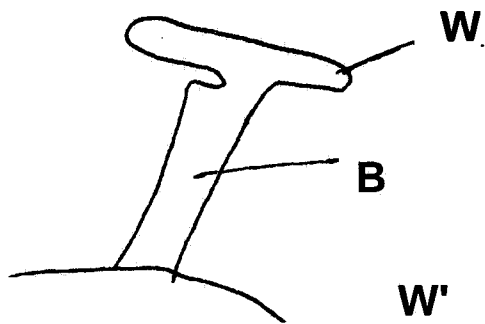


Fig. 14
(see 14-14 in Fig. 13)

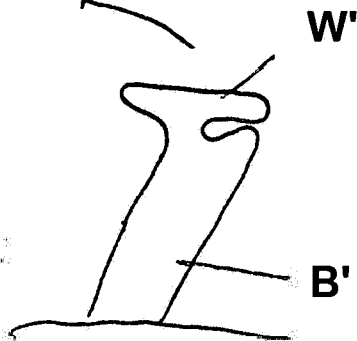


Fig. 15
(see 14-14 in Fig. 13)