



US005924413A

United States Patent [19]
Johnson et al.

[11] **Patent Number:** **5,924,413**
[45] **Date of Patent:** **Jul. 20, 1999**

- [54] **RAPID FIRE COMPRESSED AIR TOY GUN**
- [75] Inventors: **Lonnie G. Johnson**, Smyrna; **John T. Applewhite**, Atlanta, both of Ga.
- [73] Assignee: **Johnson Research & Development Company, Inc.**, Smyrna, Ga.
- [21] Appl. No.: **08/822,008**
- [22] Filed: **Mar. 24, 1997**

Related U.S. Application Data

- [63] Continuation-in-part of application No. 08/730,619, Oct. 21, 1996, Pat. No. 5,709,199, which is a continuation-in-part of application No. 08/699,431, Aug. 19, 1996, Pat. No. 5,699,781, which is a continuation-in-part of application No. 08/494,407, Jun. 26, 1995, Pat. No. 5,592,931, which is a continuation-in-part of application No. 08/441,229, May 15, 1995, Pat. No. 5,596,978.
- [51] **Int. Cl.⁶** **F41B 11/00**
- [52] **U.S. Cl.** **124/72; 124/73; 124/75; 124/76**
- [58] **Field of Search** **124/72, 73, 75, 124/76, 69**

References Cited

U.S. PATENT DOCUMENTS

2,147,003	2/1939	Von Kozurik	124/11
2,312,244	2/1943	Feltman	124/11
2,357,951	9/1944	Hale	124/11
2,654,973	10/1953	Lemelson	46/56
2,733,699	2/1956	Krinsky	124/13
2,927,398	3/1960	Kaye et al.	46/74
2,962,017	11/1960	Horowitz et al.	124/27
3,025,633	3/1962	Kaye et al.	46/74
3,121,292	2/1964	Butler et al.	46/74
3,308,803	3/1967	Walther	124/73
3,397,476	8/1968	Weber	43/6
3,612,026	10/1971	Vadas et al.	124/11

3,962,818	6/1976	Pippin	46/74
4,073,280	2/1978	Koehn	124/72
4,083,349	4/1978	Clifford	124/72
4,159,705	7/1979	Jacoby	124/63
4,223,472	9/1980	Feket et al.	46/44
4,411,249	10/1983	Fogarty et al.	124/64
4,890,767	1/1990	Burlison	222/78
4,897,065	1/1990	Fertig et al.	446/63
5,090,708	2/1992	Gerlitz et al.	273/310
5,188,557	2/1993	Brown	446/212
5,280,778	1/1994	Kotsiopoulos	124/73
5,280,917	1/1994	Ortiz	273/318
5,339,791	8/1994	Sullivan	124/73
5,343,849	9/1994	Steer	124/72
5,343,850	9/1994	Steer	124/64
5,349,938	9/1994	Farrell	124/73
5,370,278	12/1994	Raynie	222/175
5,373,832	12/1994	D'Andrade	124/69
5,398,873	3/1995	Johnson	239/99
5,415,152	5/1995	Adamson et al.	124/59
5,497,758	3/1996	Dobbins et al.	124/73
5,613,483	3/1997	Lukas et al.	124/73
5,673,679	10/1997	Walters	124/53.5
5,704,342	1/1998	Gibson et al.	124/73
5,769,066	6/1998	Schneider	124/75
5,771,875	6/1998	Sullivan	124/72

FOREIGN PATENT DOCUMENTS

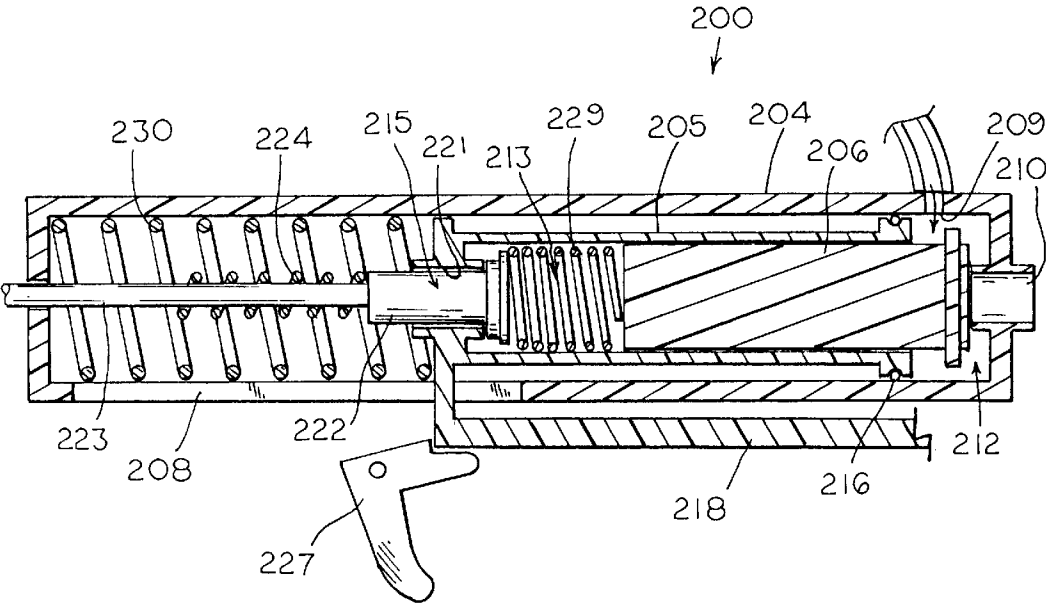
2587911-A1 10/1985 France .

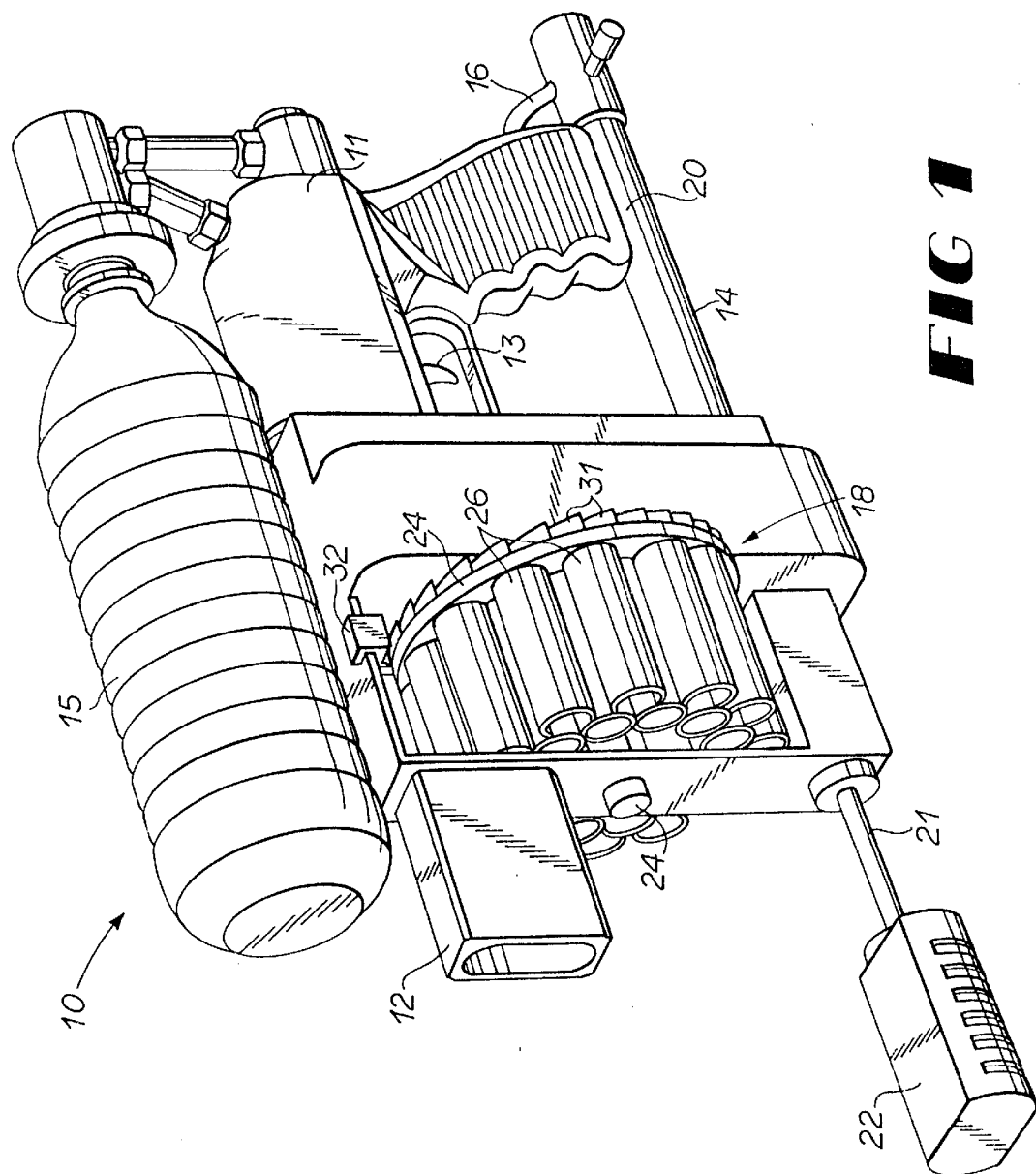
Primary Examiner—J. Woodrow Eldred
Attorney, Agent, or Firm—Kennedy, Davis & Kennedy, P.C.

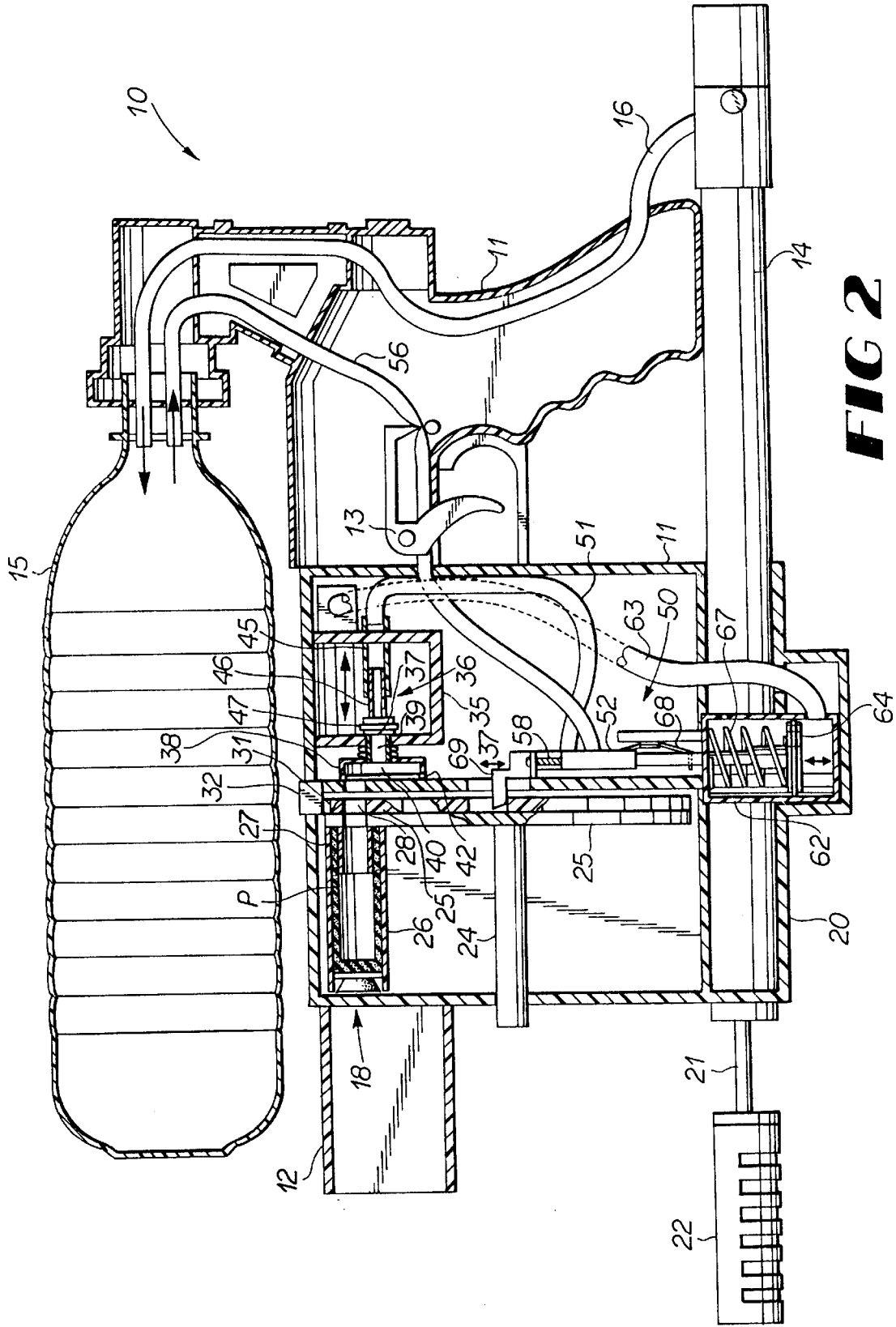
[57] **ABSTRACT**

An air compressed gun (10) is provided having a stock (11), a barrel (12), a trigger (13) and a manual air pump (14). The gun also has a control valve (200) which controls the flow of compressed air into a magazine (202). The control valve includes an external tube (204), an internal tube (205) reciprocally mounted within the external tube, and a plunger (206) reciprocally mounted within the internal tube.

31 Claims, 12 Drawing Sheets







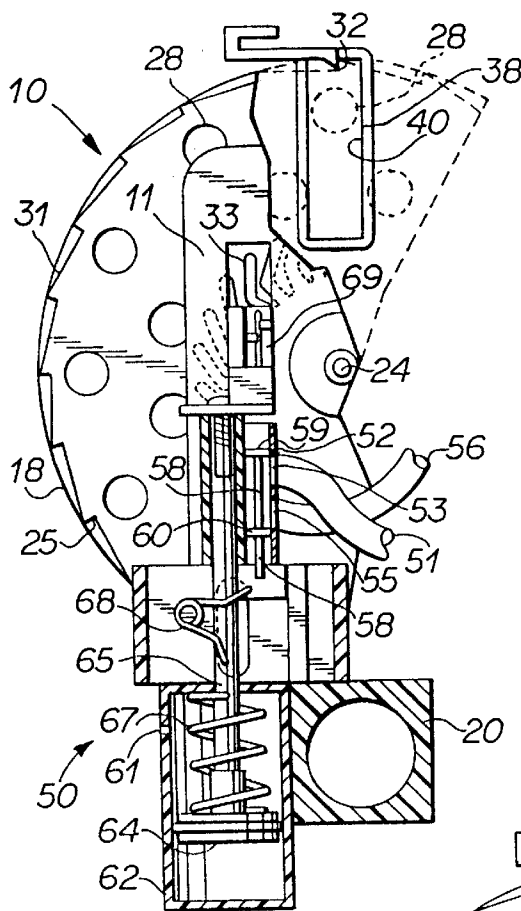


FIG 3

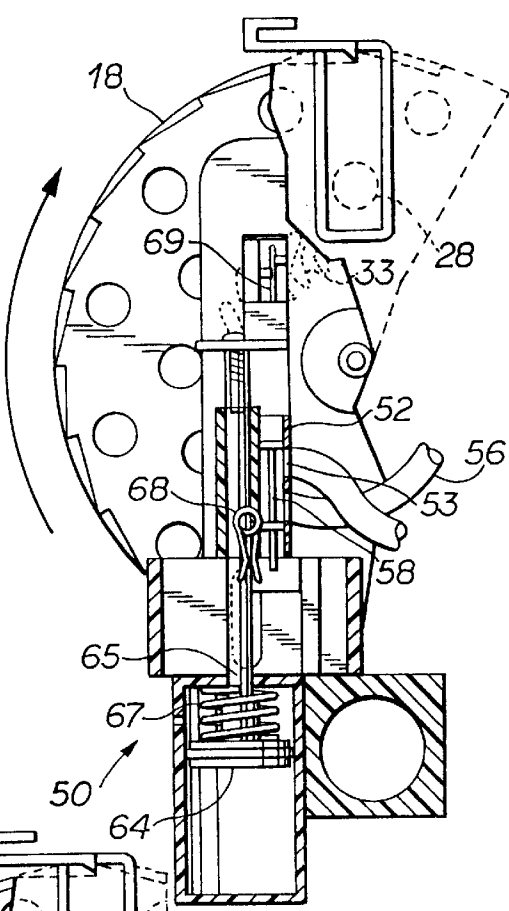


FIG 4

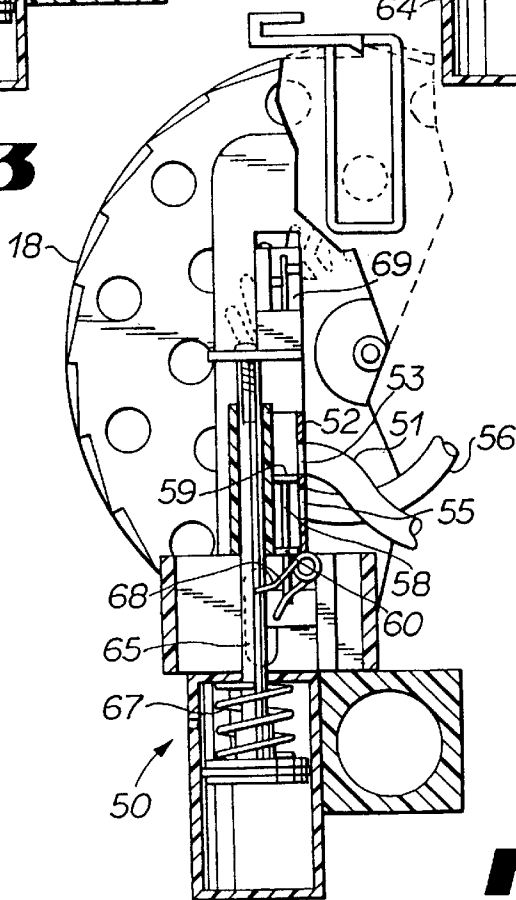
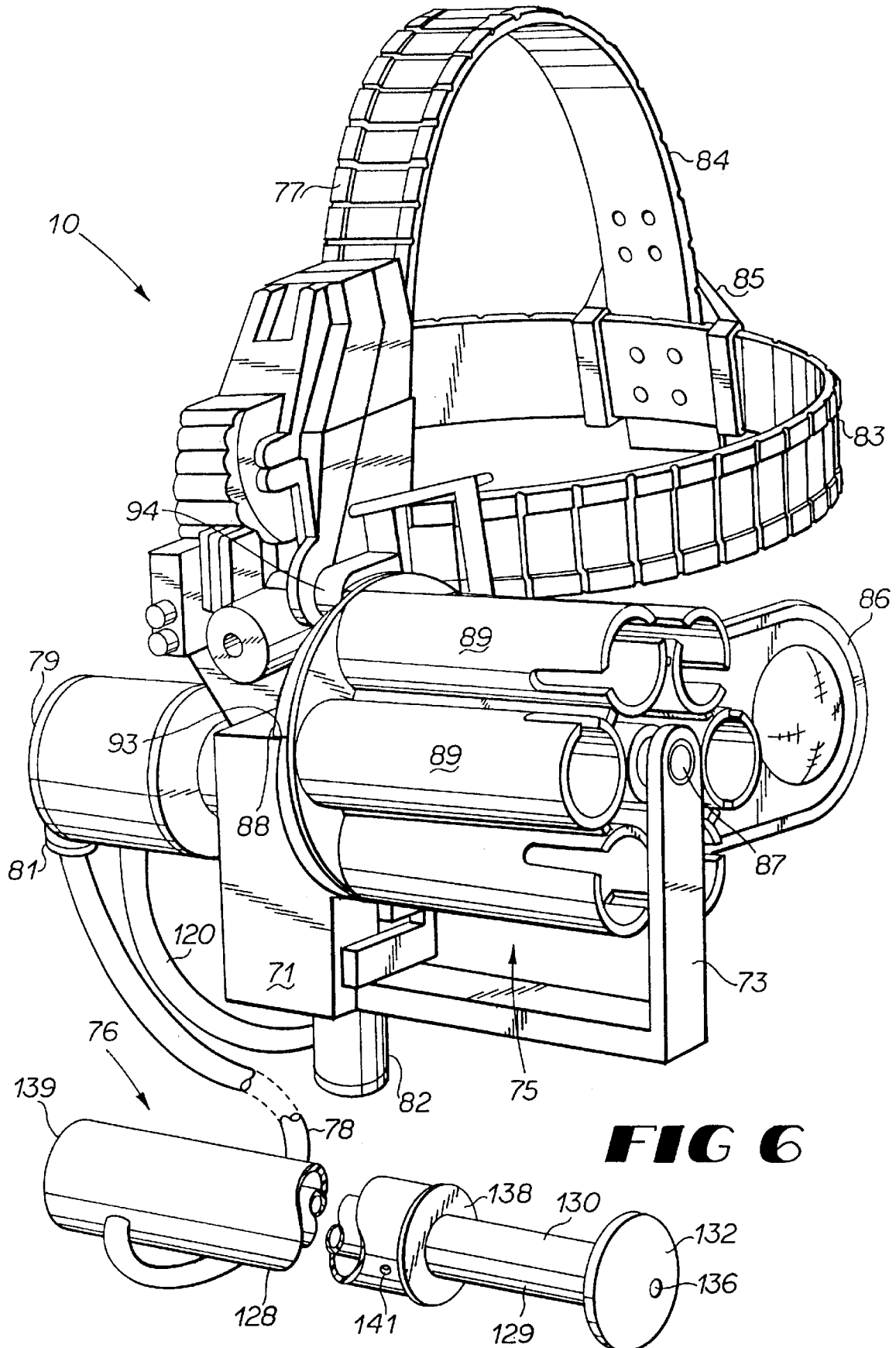
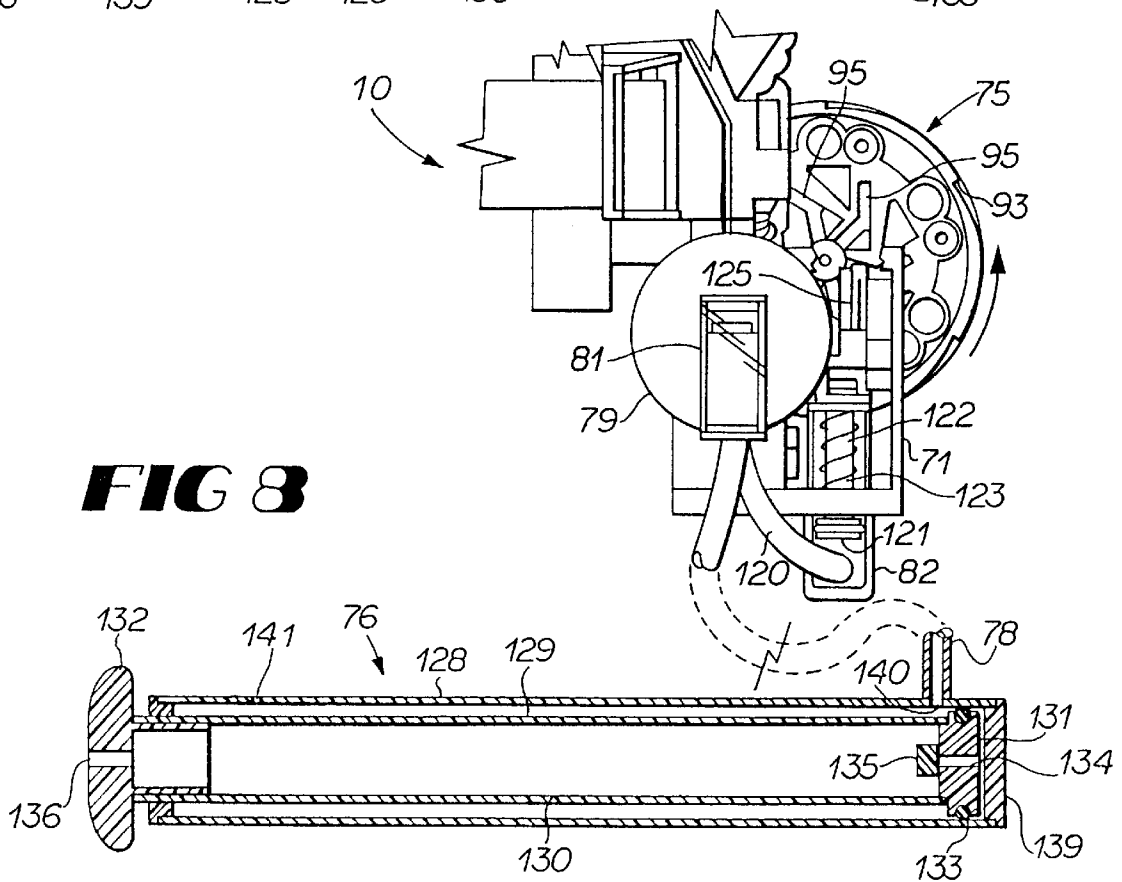
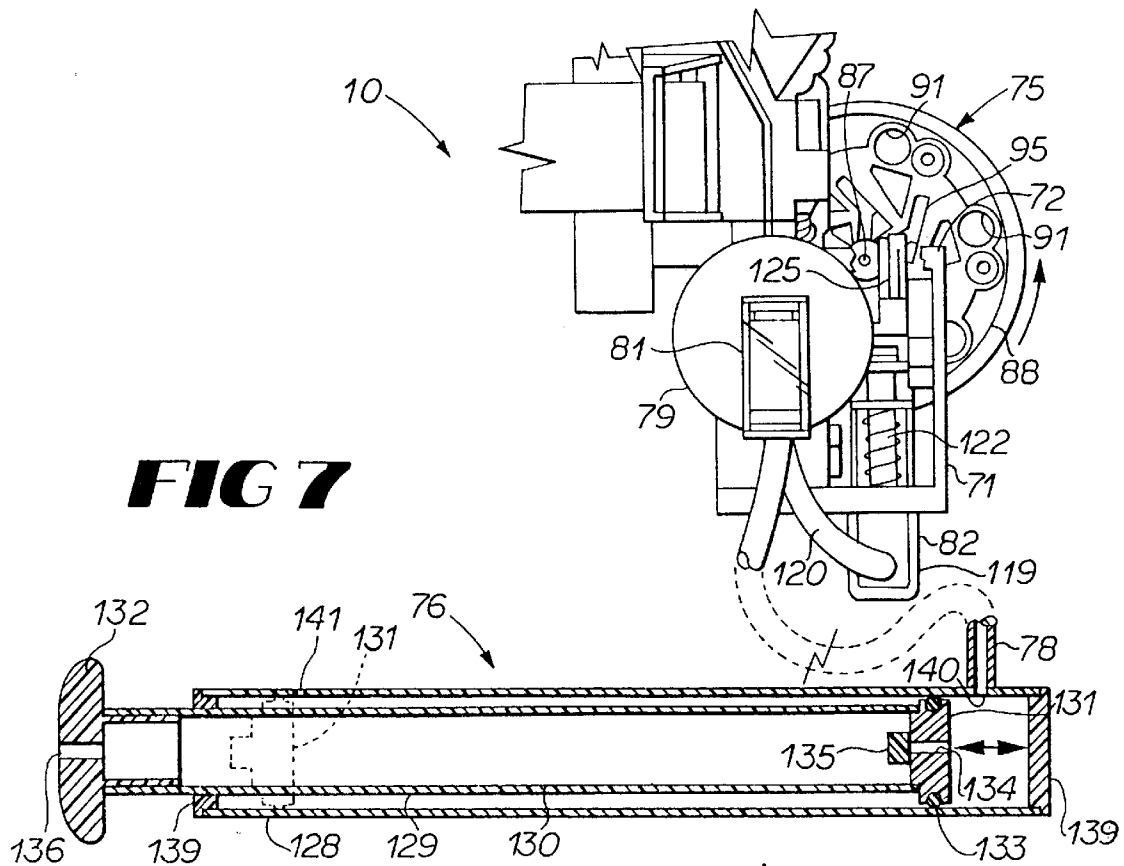


FIG 5





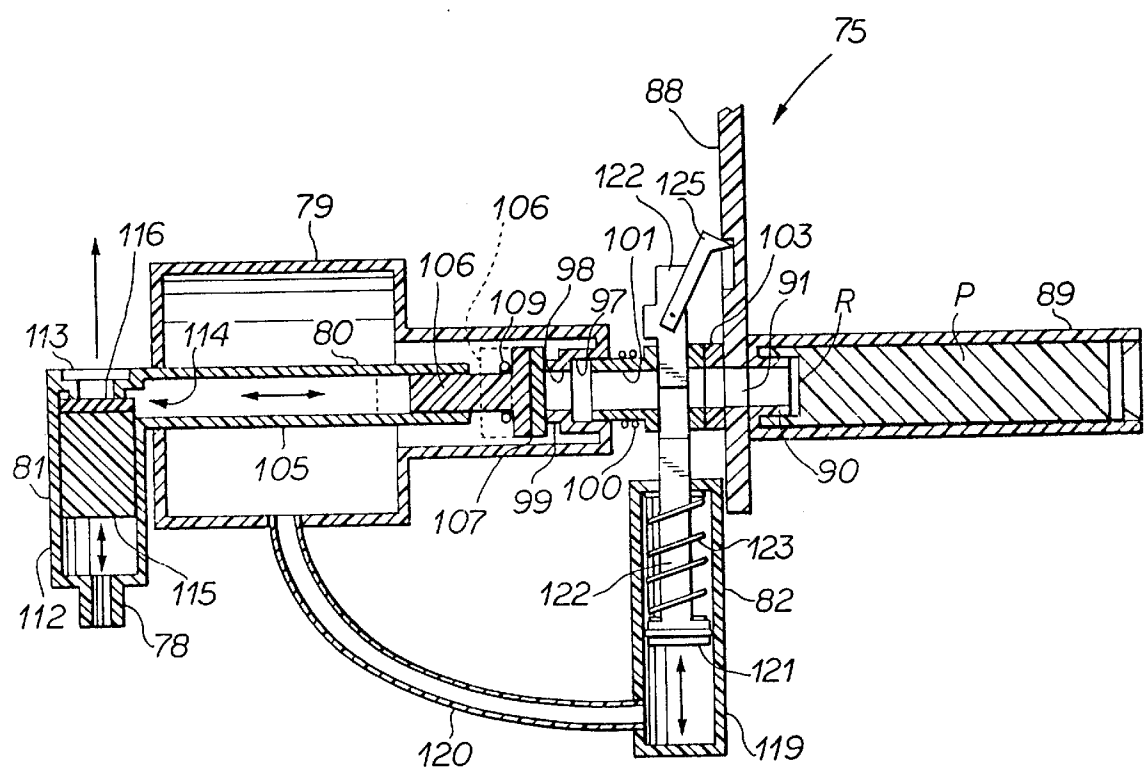
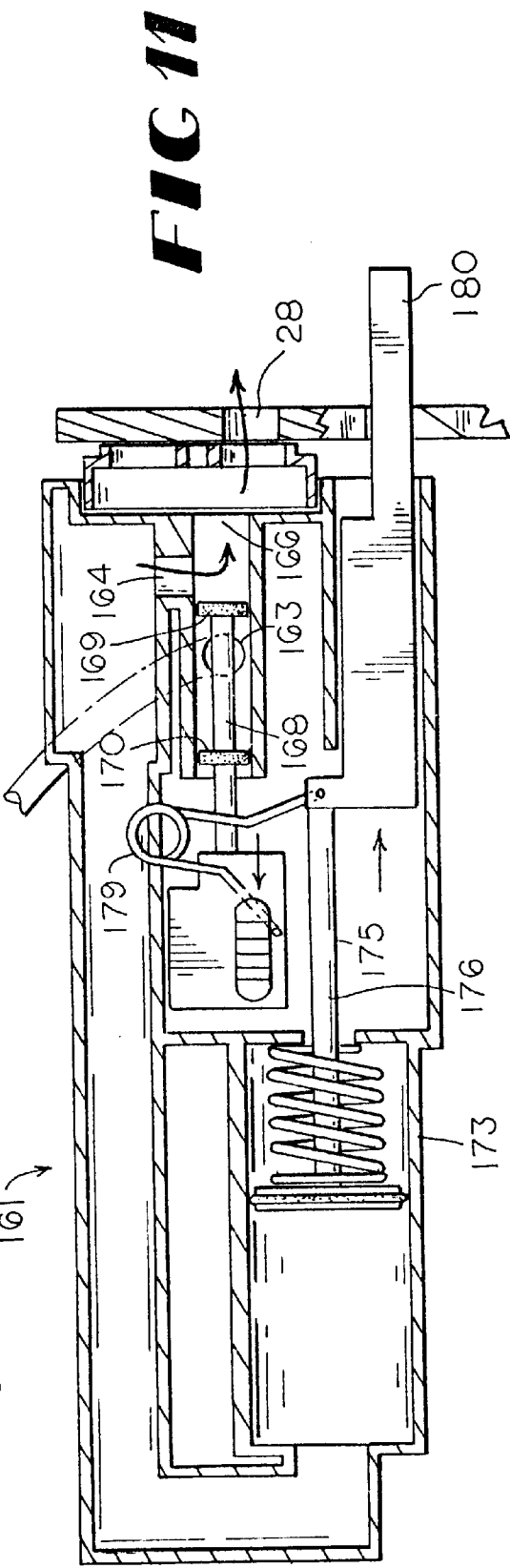
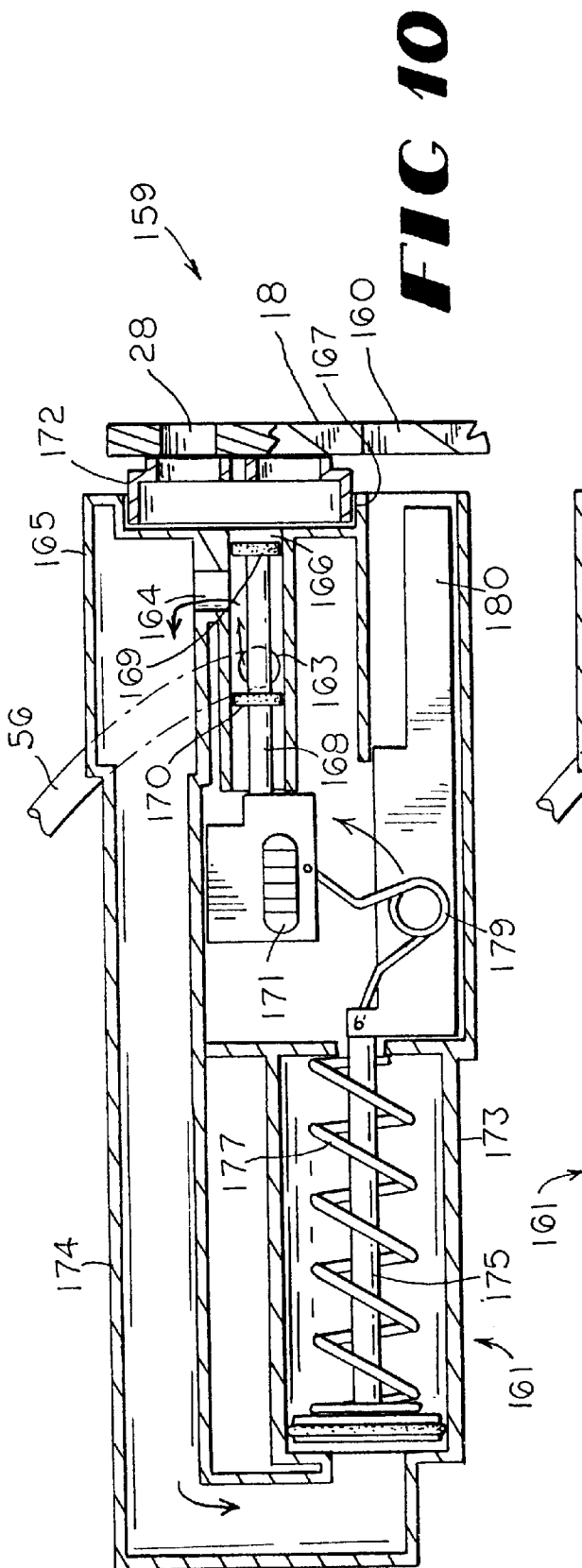


FIG 9



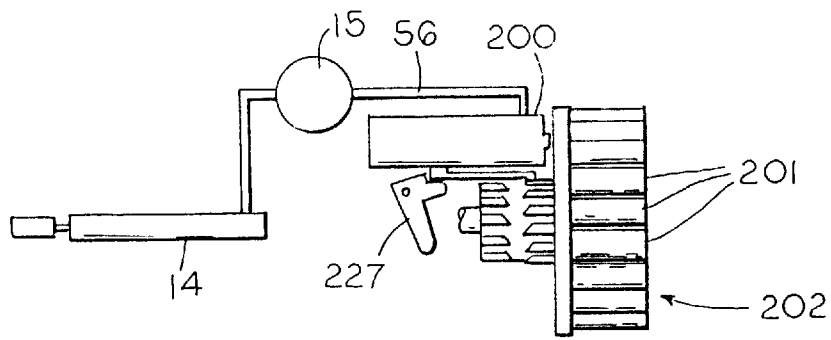


FIG 12

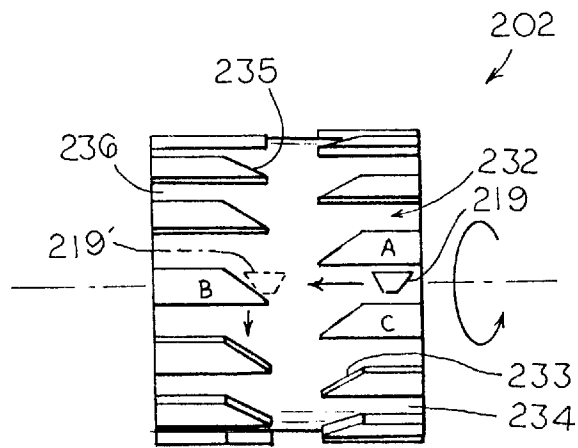


FIG 21

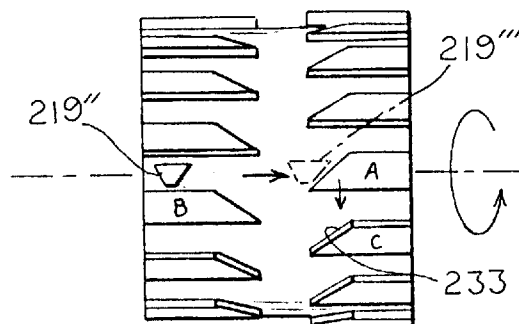


FIG 22

FIG 13

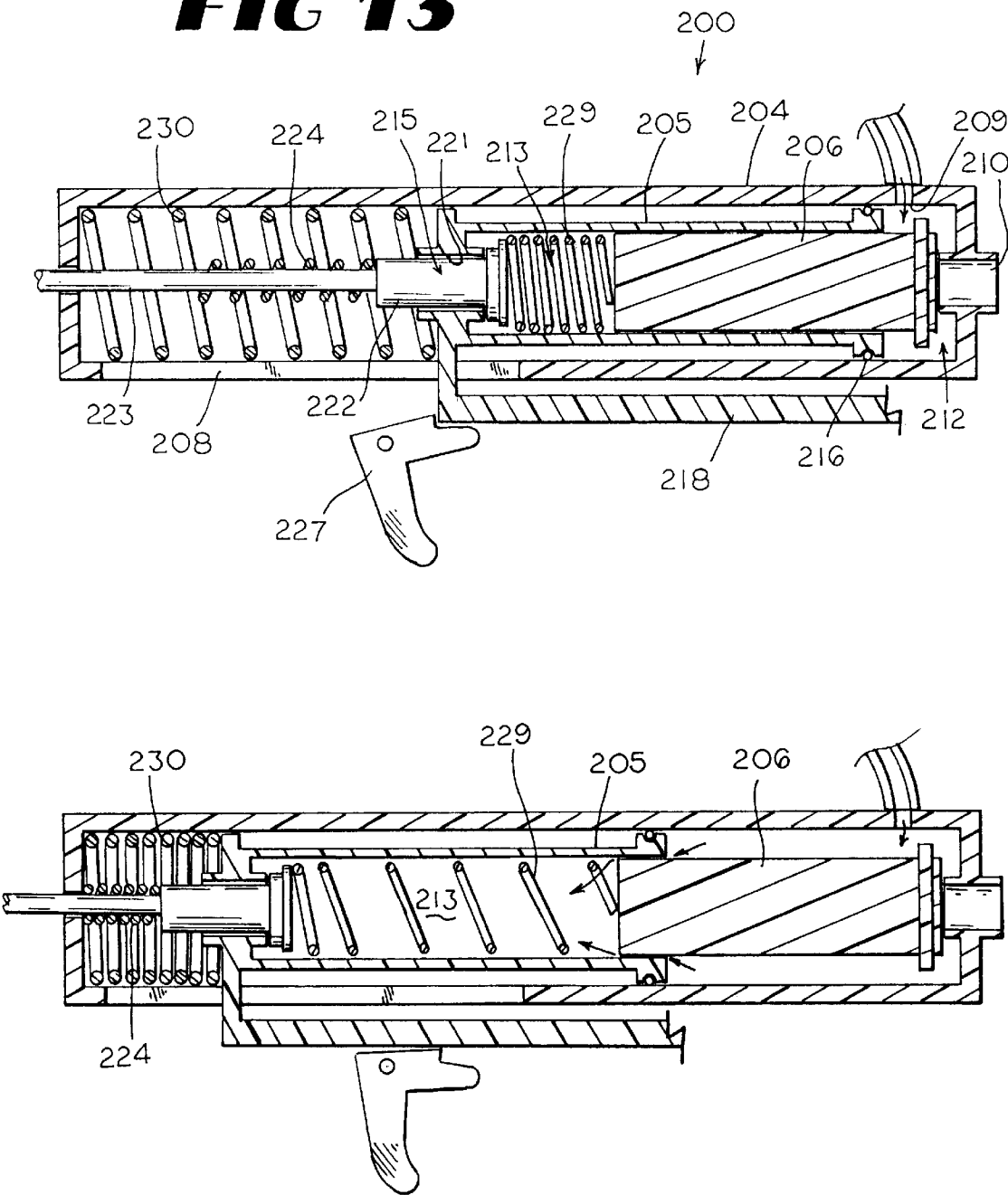


FIG 14

FIG 15

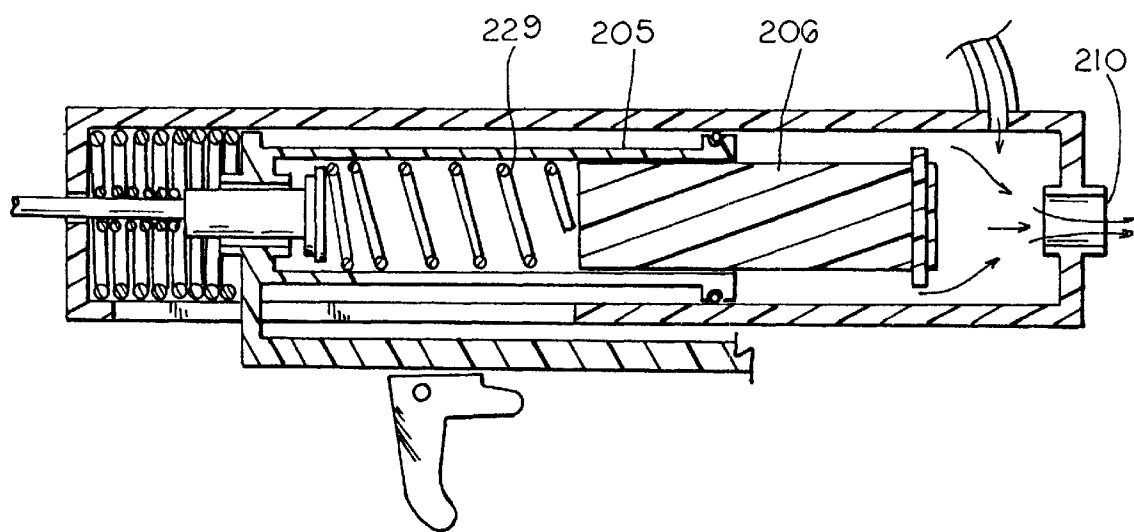
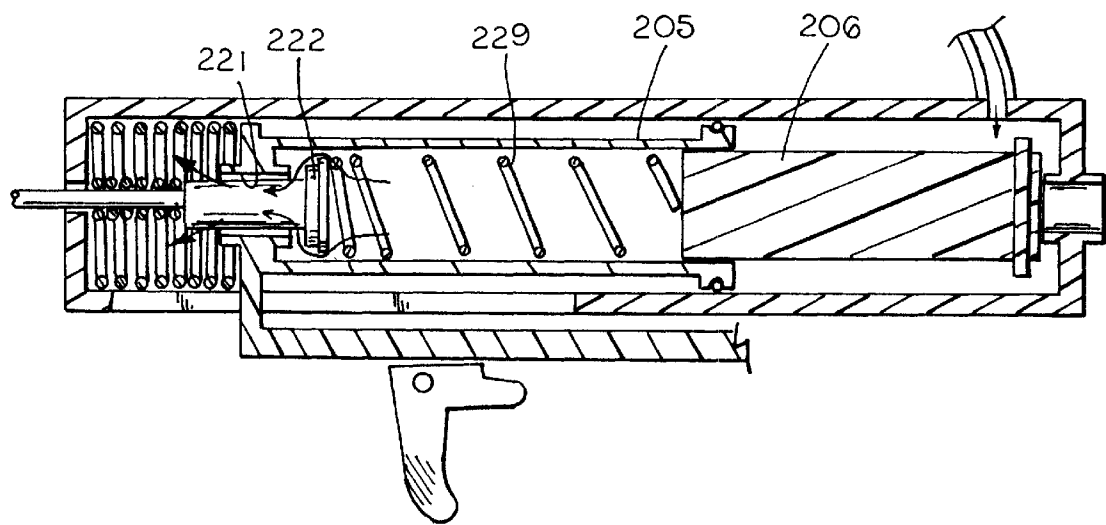


FIG 16

FIG 17

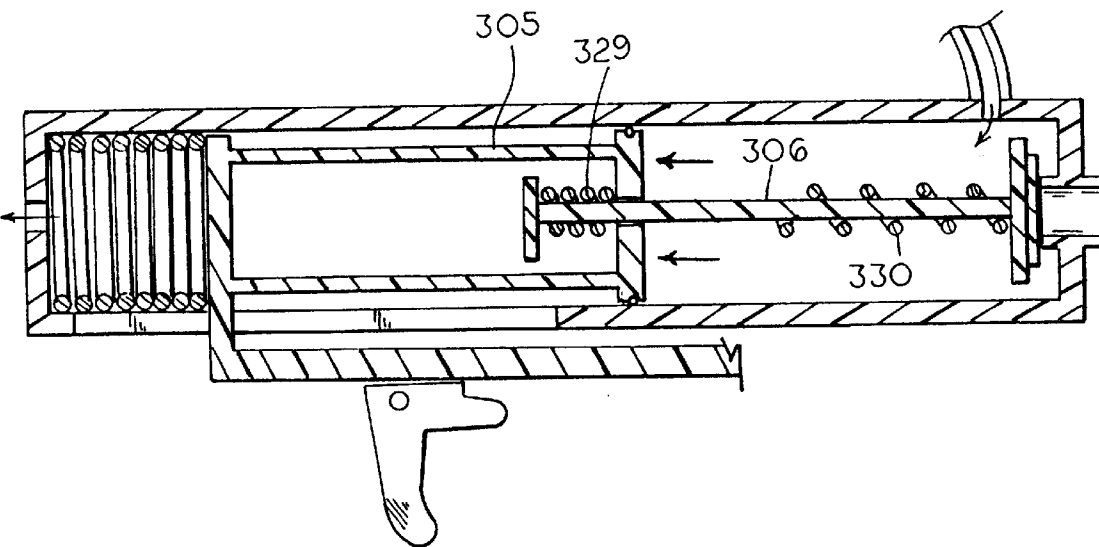
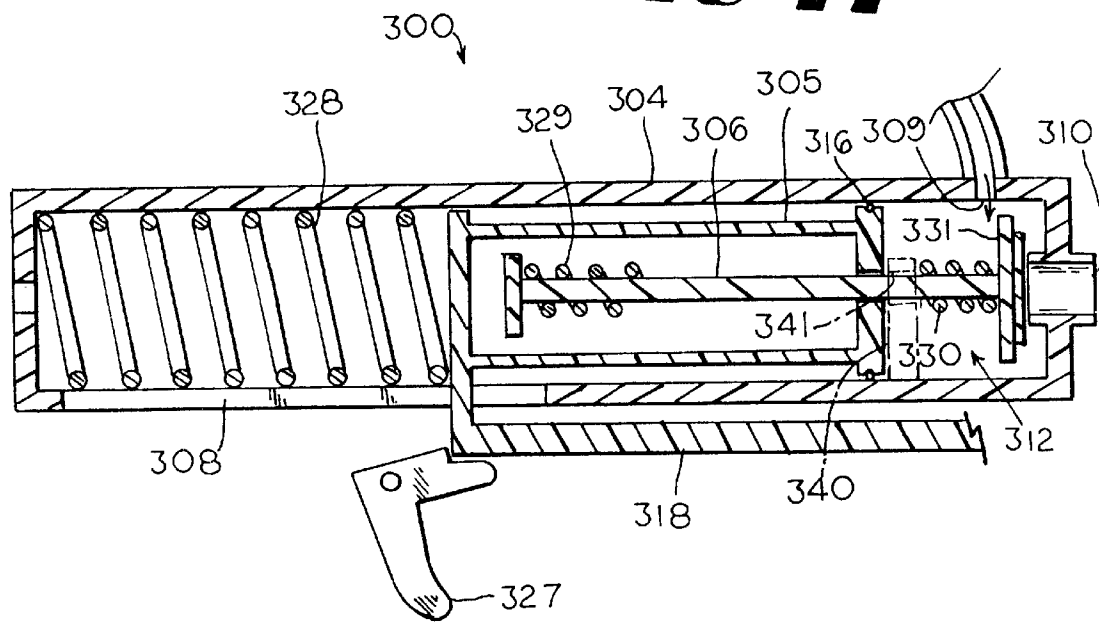


FIG 18

FIG 19

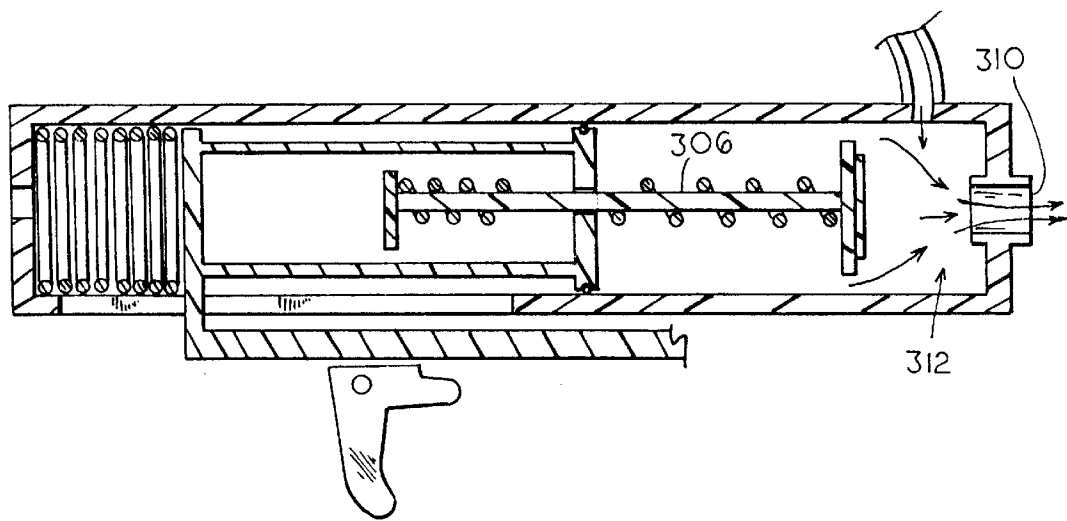
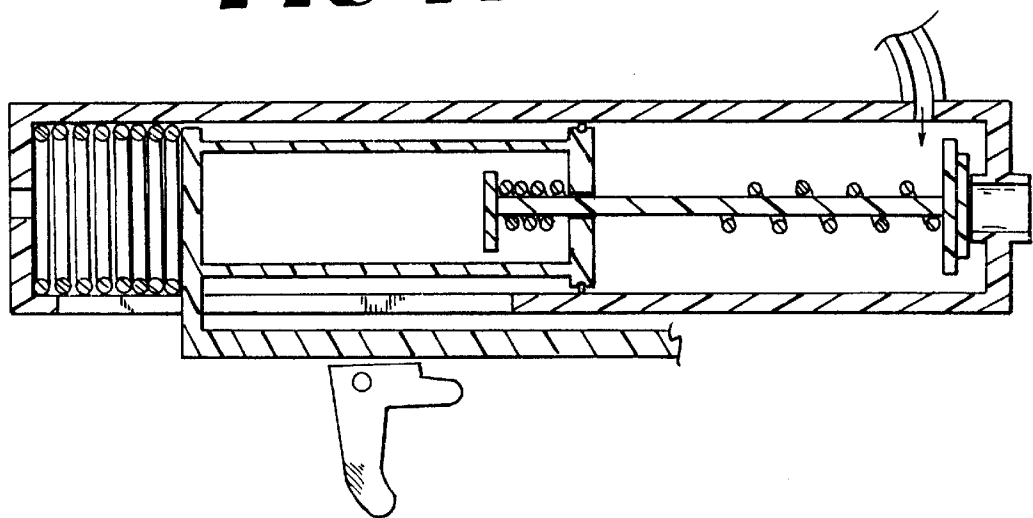


FIG 20

RAPID FIRE COMPRESSED AIR TOY GUN**REFERENCE TO RELATED APPLICATION**

This is a continuation-in-part of application Ser. No. 08/730,619 filed Oct. 21, 1996 now U.S. Pat. No. 5,709,199, which is a continuation-in-part of application Ser. No. 08/699,431 filed Aug. 19, 1996 now U.S. Pat. No. 5,699,781, which is a continuation-in-part of application Ser. No. 08/494,407 filed Jun. 26, 1995 now U.S. Pat. No. 5,592,931, which is a continuation-in-part of application Ser. No. 08/441,229 filed May 15, 1995 now U.S. Pat. No. 5,596,978.

TECHNICAL FIELD

This invention relates to compressed air guns, and specifically to compressed air toy guns which include a magazine for holding projectiles and an indexer for indexing the magazine.

BACKGROUND OF THE INVENTION

Toy guns which shoot or launch projectiles have been very popular for many years. These guns have been designed to launch projectiles in a number of ways. A common method of launching has been by the compression of a spring which propels the projectile upon its decompression or release, as, for example, with BB guns and dart guns. These guns however usually do not generate enough force to launch projectiles with great velocity.

Toy guns have also been designed which use compressed air to launch projectiles such as foam darts. These types of guns use a reciprocating air pump to pressurize air within a pressure tank. In use, a single dart is loaded and the pump is typically reciprocated several times with each firing of the gun. Therefore, the gun must be loaded and pumped with each firing as it is not capable of firing several darts in rapid sequence. The rapid firing of a gun may be desired for those playing a mock war or other type of competition.

As children often become bored with the design of conventional guns it is desirable to design guns having an unconventional construction or appearance. However, unconventional guns are often difficult to accurately aim and fire.

Today children who play mock wars often carry several guns at one time in order to fire several shots simultaneously or in rapid succession. This however is difficult as two hands must be used to fire two separate guns and two hands are typically used to pump one gun. Hence, a child must choose to either fire a gun in each hand or pump one gun for firing.

Accordingly, it is seen that a need remains for a toy air gun which may be fired without restricting an operator's hands. Also, it is seen that a need remains for a toy air gun of an unconventional design which may be accurately aimed and fired. It is to the provision of such therefore that the present invention is primarily directed.

SUMMARY OF THE INVENTION

In a preferred form of the invention a compressed air toy gun for firing projectiles comprises pump means for compressing air, launch tube means for holding a projectile, conduit means for conveying compressed air from the pump means to the launch tube means, and control valve means for controlling the flow of compressed air from the pump means to the launch tube means. The control valve means has an external housing having an air inlet in fluid communication with the conduit means and an air outlet in fluid communication with the launch tube means. The control valve means

also has an internal housing and a plunger. The internal housing is mounted for reciprocal movement within the external housing between a first position and a second position. Internal housing biasing means biases the internal housing towards the first position. The plunger is mounted within the internal housing for reciprocal movement between a sealing position sealing the air outlet and an unsealing position unsealing the air outlet. Plunger biasing means biases the plunger toward the sealing position. Actuation means actuates the movement of the plunger to the unsealing position. The external housing and the internal housing at least partially defining an air pressure chamber in fluid communication with the air inlet.

With this construction compressed air from the pump means is conveyed through the conduit means and through the air inlet into the air pressure chamber. The compressed air within the air pressure chamber biases the internal housing to its second position whereupon the actuation means causes the movement of the plunger is moved to its unsealing position allowing the compressed air within the air pressure chamber to flow through the air outlet to the launch tube.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a perspective view of a rapid fire compressed air gun embodying principles of the present invention in a preferred form.

FIG. 2 is a side view, shown in partial cross-section, of the air gun of FIG. 1.

FIGS. 3-5 are a sequence of views showing a portion of the air gun of FIG. 1, which show in sequence, the actuation of an actuator which indexes a magazine and controls a release valve.

FIG. 6 is a perspective view of a rapid fire compressed air gun embodying principles of the present invention in another preferred form.

FIG. 7 is a rear view of portions of the air gun of FIG. 6 with the pump shown in side view for clarity of explanation.

FIG. 8 is a rear view of portions of the air gun of FIG. 6 with the pump shown in side view for clarity of explanation.

FIG. 9 is a side view, shown in partial cross-section, of interior components of the air gun of FIG. 6 and a projectile positioned within the barrel of the gun.

FIG. 10 is a side view, shown in partial cross-section, of an alternative design for the interior components of the air gun of FIG. 1, shown in a pressurizing configuration.

FIG. 11 is a side view, shown in partial cross-section, of the interior components shown in FIG. 10, shown in a firing configuration.

FIG. 12 is a schematic view of portions of an air compressed gun in another preferred form.

FIGS. 13-16 are a sequence of side views, shown in partial cross-section, of a portion of the interior components of the air gun of FIG. 12, which show in sequence, the actuation of the interior components controlling the release of pressurized air.

FIGS. 17-20 are a sequence of side views, shown in partial cross-section, of a portion of the interior components in another preferred embodiment, which show in sequence, the actuation of the interior components controlling the release of pressurized air.

FIGS. 21 and 22 are a sequence of top views of the magazine of the air gun of FIG. 12, which show in sequence, the rotation of the magazine in conjunction with the actuation of the control valve.

DETAILED DESCRIPTION

With reference next to the drawings, there is shown a compressed air gun 10 having a stock or handle 11, a barrel 12 mounted to the stock 11, a spring biased trigger 13, and a manual air pump 14. The gun 10 has a pressure chamber or tank 15 in fluid communication with the air pump 14 through a pressure tube 16 and a multi-projectile magazine 18 rotationally mounted to stock 11. The pump 14 includes a conventional cylinder 20, a cylinder rod 21 and a handle 22 mounted to an end of the cylinder rod 21.

The magazine 18 has a central pivot rod 24 mounted to a diskshaped mounting plate 25 and an annular array of projectile barrels 26 extending from the mounting plate 25 in generally two concentric circles about pivot rod 24. Each barrel 26 has a launch tube 27 therein aligned with an opening 28 extending through the mounting plate 25. Likewise, the openings 29 are oriented in two concentric circles or annular arrays with each opening of the inner circle being positioned generally between two adjacent opening of the outer circle, so as to appear in staggered fashion, as best shown in FIGS. 3-5. Thus, each opening 28' of the outer annular array of openings 28' is aligned along a radius and spaced a selected distance d1 from the center of the mounting plate, and each opening 28" of the inner annular array of openings 28" is aligned along a radius and spaced a selected distance d2 from the center. The gun magazine is shown in FIG. 2 as having only one barrel for clarity of explanation. Mounting plate 25 has series of peripheral, outwardly extending, serrated teeth 31 each of which is aligned with a barrel 26. The serrated teeth 31 are configured to cooperate with a pawl 32 extending from the stock 11. The mounting plate 25 also has an annular array of L-shaped grooves 33 equal in number to the number of magazine barrels 26.

The gun 10 has a pressure chamber 35 adapted to receive and store a supply of air at elevated pressure levels and a pressure sensitive release valve 36 mounted within the pressure chamber 35. The pressure chamber 35 has an exit opening 37 therein. A spring biased sealing plate 38 is mounted within opening 37. The sealing plate 38 has a central bore 39 extending into an elongated bore 40 configured to overlay the mounting plate openings 28. It should be noted that the mounting plate openings 28 are positioned so that the sealing plate elongated bore 40 overlaps only one opening 28 at a time. A gasket 42 is mounted to the sealing plate 38 to ensure sealing engagement of the sealing plate with the mounting plate 25. The release valve 36 has a cylindrical manifold 45 and a cylindrical plunger 46 slidably mounted within manifold 45. Plunger 46 has a gasket 47 to ensure sealing engagement of the plunger about opening 37.

The release valve manifold 45 is pneumatically coupled to an actuator 50, by a pressure tube 51 extending therebetween the actuator 50 automatically and sequentially causes the actuation of the release valve 36. Actuator 50 includes an elongated manifold 52 having an upper opening 53 in fluid communication with pressure tube 51 and a lower opening 55 in fluid communication with another pressure tube 56 extending from the pressure tank 15 and positioned so as to be pinchably closed by spring biased trigger 13. A piston 58 is movably mounted within actuator manifold 52. Piston 58 has a top seal 59 and a bottom seal 60. The actuator 50 also has a pressure cylinder 62 having a vent 61 adjacent its top end. Pressure cylinder 62 is coupled in fluid communication with pressure chamber 35 by a pressure tube 63. A piston 64, having an elongated piston rod 65, is mounted within the actuator pressure cylinder 62 for reciprocal movement

therein between a low pressure position shown in FIGS. 2 and 3 and a high pressure position shown in FIG. 4. A coil spring 67 mounted about piston rod 65 biases the piston 64 towards its low pressure position. Piston rod 65 is coupled to piston 58 by an over center torsion spring 68, such as that made by Barnes Group Incorporated of Corry, Pennsylvania under model number T038180218-R. An indexing finger 69, mounted to an end of the piston rod 65, is configured to sequentially engage and ride within each magazine L-shaped groove 33.

In use, an operator actuates the pump to pressurize a supply of air by grasping the handle 22 and reciprocating the cylinder rod 21 back and forth within the cylinder 20. Pressurized air is passed through pressure tube 16 into the pressure tank 15. Manual actuation of the trigger 13 moves the trigger to a position wherein it unpinches pressure tube 56 so as to allow pressurized air within the pressure tank 15 to pass through pressure tube 56 into actuator manifold 52 between the top and bottom seals 59 and 60. The pressurized air then passes out of lower opening 55 and through pressure tube 51 into release valve manifold 45.

The pressurized air within the release valve manifold 45 causes the plunger 46 to move to a forward position sealing the opening 37. Pressurized air then flows between the plunger 46 and the release valve manifold 45 so as to pressurize the pressure chamber 35. A portion of the pressurized air within pressure chamber 35 passes through pressure tube 63 into the actuator pressure cylinder 62. With increased pressure within pressure cylinder 62 the piston 64 is forced upwards against the biasing force of coil spring 67, i.e. the piston 64 is moved from its low pressure position shown in FIG. 3 to its high pressure position shown in FIG. 4. As shown in FIG. 4, upward movement of the piston rod 65 causes compression of torsion spring 68 and the finger 69 to ride up within a mounting plate groove 33 thereby causing clockwise rotation of the magazine 18 which brings opening 28" into fluid communication with seal plate 38. All references herein to downward and upward directions is for purposes of clarity in reference to the drawings and is not meant to indicate gravity sensitivity. Upon reaching the apex of the movement of piston rod 65 the torsion spring 68 decompresses thereby forcing piston 58 downward, as shown in FIG. 5. Downward movement of piston 58 causes the top seal 59 to be positioned between upper opening 53 and lower opening 55. This positioning of the piston 58 isolates manifold lower opening 55 to prevent escape of pressurized air from pressure tank 15. This positioning of the top seal 59 also allows pressurized air within pressure tube 51 to escape to ambience through the top of actuator manifold 52. The release of air pressure causes the plunger 46 to move to a rearward position unsealing opening 37. With the unsealing of opening 37 pressurized air within pressure chamber 35 flows through opening 37, into the central and elongated bores 39 and 40 of sealing plate 38, and into the launch tube 27 through mounting plate opening 28. Pressurized air within launch tube 27 propels the projectile out of the magazine barrel 26 and through gun barrel 12. The actuation of this type of release valve is described in more detail in U.S. Pat. No. 4,159,705.

Upon the release of pressurized air from pressure chamber 35 the pressurized air within pressure cylinder 62 is released through pressure tube 63 back into pressure chamber 35. The release of air from pressure cylinder 62 causes the piston 64 be spring biased by coil spring 67 back downward to its low pressure position. The downward movement of piston 64 retracts the indexing finger 69 from within a mounting plate groove 33 and positions the finger in register with the

following mounting plate groove 33. The low pressure positioning of piston 64 causes the torsion spring 68 to bias piston 58 upwards to its initial position with the top and bottom seals 59 and 60 straddling upper and lower openings 53 and 55, as shown in FIG. 3. This repositioning of piston 58 once again causes pressurized air within pressure tank 15 to flow through pressure tube 56 into actuator manifold 52, thereby completing a firing cycle. The firing and indexing cycle just describe may continue in rapid sequence so long as the trigger is maintained in a position allowing the flow of pressurized air through pressure tube 56 and the pressure tank continues to contain a minimal level of pressurized air sufficient to overcome the biasing force of springs 67 and 68, i.e. the release valve is automatically actuated by actuator 50 and the indexing of magazine 18 continues so long as the trigger is pulled open and the pressure tank contains pressurized air above a level to overcome springs 67 and 68. Should the pressure level within pressure tank 15 reach the minimal level the operator simply actuates the manual air pump 14 so as to once again elevate the pressure within the pressure tank.

As described, the gun may be used in a fully automatic manner such that with the trigger maintained in a pulled back, actuated position the gun fires a series of projectiles without stopping between each successive shot, similar to the action of a machine gun. However, should an operator wish to fire a single projectile, one need only to pull the trigger and quickly release it so that pressurized air does not continue to flow into the actuator 50. Operated in such a manner the gun will index the magazine and fire a projectile with each actuation of the trigger, again, so long as the pressure tank contains air pressurized above the minimal level and the trigger is quickly released.

It should be noted that pawl 32 engages teeth 31 to prevent rotation of the magazine in a direction opposite to its indexing direction, i.e. to prevent counterclockwise rotation in FIG. 3. This prevents the firing of pressurized air into a just emptied barrel and damage to the indexing finger. It should also be noted that since the pneumatic system is closed, once the gun is initially pressurized it is maintained under at least the minimal pressure level. Thus, the gun has the capability of firing projectiles in a rapid sequence of shots one after another. Yet, the gun may also fire a sequence of single shots without having to be pumped between each successive shot.

Referring next to FIGS. 6-9, a compressed air gun 70 in another preferred form is shown. Here, the air gun 70 has a housing 71 having a support plate 72 and an L-shaped support arm 73, a magazine 75 rotationally mounted to the housing 71, a remote manual hand air pump 76, and a harness 77 secured to housing 71 and configured to be supported upon the head of a person. The gun 70 has a pressure chamber 79 adapted to receive and store a supply of air at elevated pressure levels and a pressure actuable release valve 80 mounted within the pressure chamber 79. A control valve 81 is mounted in fluid communication with release valve 80 and is coupled in fluid communication with pump 76 by a pressure tube 78 extending therebetween. Pressure chamber 79 is pneumatically coupled to a pneumatic indexer 82 which in turn is coupled to magazine 75 for rotational movement thereof.

The head harness 77 has a generally circular base strap 83 and a inverted U-shaped, adjustable top strap 84 secured to the base strap 83 by a buckle 85. The head harness 77 also has a clear eye sight 86 configured to be positioned over the eye of a person. The top strap 84 and base strap 83 may be made of a soft, flexible plastic which can conform to the person's head.

The magazine 75 has a central pivot rod 87 fixedly mounted to a disk-shaped mounting plate 88 and an annular array of projectile barrels or launch tubes 89 extending from the mounting plate 88 in a generally concentric circle about pivot rod 87. Pivot rod 87 is rotationally mounted at one end to support arm 73 and rotationally mounted at its opposite end to support plate 72. Each barrel 89 has a launch tube 90 therein aligned with an opening 91 which extends through the mounting plate 88. The interior diameter of barrel 89 is configured to releasably hold a projectile P with the launch tube 90 configured to be received within a recess R in the rear of the projectile. The magazine is shown in FIG. 9 as having only one barrel 89 for clarity of explanation. Mounting plate 88 has series of peripheral notches 93 each of which is aligned with a barrel 89. The notches 93 are configured to cooperate with a pawl 94 extending from the housing 71. Mounting plate 88 also has an annular array of L-shaped grooves 95 oriented about pivot rod 87 which are equal in number to the number of magazine barrels 89.

The pressure chamber 79 has a recess 97 having an air exit opening 98 therein defined by an inwardly extending annular flange 99. A spring biased sealing plate 100 is mounted within recess 97. The sealing plate 100 has a central bore 101 configured to overlay the mounting plate openings 91 of the magazine. It should be noted that the mounting plate openings 91 are positioned so that the sealing plate bore 101 overlaps only one opening 91 at a time. A gasket 103 is mounted to the sealing plate 100 to ensure sealing engagement with the mounting plate 88. The release valve 80 has a cylindrical manifold 105 and a cylindrical plunger 106 slidably mounted within the manifold 105. Plunger 106 has a gasket 107 to ensure sealing engagement of the plunger 106 about opening 98 with the plunger in a sealing position shown in FIG. 9, and a O-ring type seal 109 to ensure sealing engagement of the plunger 106 against manifold flange 99 with the plunger in a released position shown in phantom lines in FIG. 9.

The control valve 81 has an elongated cylindrical manifold 112 having a top vent opening 113 to ambience, a side opening 114 in fluid communication with release valve manifold 105, and a cylindrical plunger 115 slidably mounted within manifold 112. Plunger 115 has a gasket 116 to ensure sealing engagement of the plunger about vent opening 113 with the plunger in a pressurized position shown in FIGS. 7 and 9.

The indexer 82 has a pressure cylinder 119 coupled in fluid communication with pressure chamber 79 by a pressure tube 120. A piston 121, having an elongated piston rod 122, is mounted within the indexer pressure cylinder 119 for reciprocal movement therein between a low pressure position shown in FIG. 8 and a high pressure position shown in FIGS. 7 and 9. A coil spring 123 is mounted about piston rod 122 so as to bias the piston 121 towards its low pressure position. A spring biased indexing finger 125 is pivotably mounted to piston rod 125. Indexing finger 125 is configured to sequentially engage and ride within each magazine groove 95 as the piston rod is moved upward and to disengage the groove as the piston rod is moved downward. All references herein to downward and upward directions is for purposes of clarity in reference to the drawings and is not meant to indicate gravity sensitivity.

The air pump 76 includes an elongated cylinder 128 and a plunger 129 telescopically mounted for reciprocal movement within the cylinder 128. Plunger 129 has a tubular shaft 130 with an enlarged sealing end 131 and a handle 132 opposite the sealing end 131. Sealing end 131 has an O-ring type seal 133 with an opening 134 therethrough, and a

conventional check valve **135** mounted within opening **134**. Check valve **135** is oriented to allow air to pass from the interior of cylinder **128** through opening **134** into the interior of shaft **130** and to prevent air from passing through opening **134** in the opposite direction. Handle **132** has a vent **136** therethrough which allows air to pass from ambience into the interior of shaft **130**.

Pump cylinder **128** has an open end **138** through which plunger **129** extends and a closed end **139**. The pump cylinder **128** also has a port **140** in fluid communication with pressure tube **78** and a vent **141** adjacent open end **138** which is open to ambience. Port **140** is spaced from closed end **139** so as to allow seal **133** of plunger **129** to be moved past the port **140** to a position closely adjacent to the closed end **139**, as shown in FIG. 8.

In use, a person dons the gun by securing the head harness **77** to his head with the magazine **75** to one side. The person then actuates the pump **76** by grasping the pump handle **132** and forcing the pump plunger **129** through cylinder **128** towards port **140** thereby pressurizing air within the cylinder. Thus, the plunger **129** is moved from a first position shown in phantom lines in FIG. 7 to generally a second position shown in FIG. 7. The pressurized air passes through port **140** into pressure tube **78** where it then passes through control valve **81**. The increase in air pressure within the control valve manifold **112** forces the control valve plunger **115** to move to an upper, pressurized position sealing vent opening **113**, as shown in FIG. 9. The pressurized air then passes about plunger **115** and through side opening **114** into the release valve manifold **105**. The increase in air pressure within the release valve manifold **105** forces the control valve plunger **106** to move to a forward, pressurized position sealing opening **98**, as shown in FIG. 9. The pressurized air then flows between the release valve plunger **106** and the release valve manifold **105** into pressure chamber **79**.

A portion of the pressurized air within pressure chamber **79** passes through pressure tube **120** into the indexer pressure cylinder **119**. With increased pressure within pressure cylinder **119** the indexer piston **121** is forced upwards against the biasing force of coil spring **123**, i.e. the indexer piston **121** is moved from its low pressure position shown in FIG. 8 to its high pressure position shown in FIGS. 7 and 9. As shown in FIG. 9, upward movement of the piston rod **122** causes the finger **125** to ride up within a mounting plate groove **95** to cause counter-clockwise rotation of the magazine **75** as indicated by arrows in FIGS. 7 and 8.

With continued movement of the pump plunger **129** within pump cylinder **128** the seal **133** passes pump cylinder port **140**, as shown in FIG. 8. With the plunger seal **133** in this position pressurized air within pressure tube **78** is released back into pump cylinder **128** behind seal **133** and then to ambience through vent **141**. The reentry of pressurized air into the pump cylinder **128** from pressure tube **78** causes the control valve plunger **115** to move to a downward position unsealing vent opening **113**, as shown in FIG. 8. Thus, the decrease in air pressure within the pressure tube **78** and control valve manifold **112** triggers the actuation of control valve **81** to its open configuration. The actuation of the control valve to its open, downward position causes a release of pressurized air from within release valve manifold **105** through the control valve side opening **113** and then through vent opening **113** to ambience. This decrease in pressure causes release valve plunger **106** to move to a rearward position unsealing opening **98**, as shown in phantom lines in FIG. 9. The position of the plunger **106** also causes and the O-ring to abut manifold **105** to seal the path between the manifold **105** and plunger **106**. With the unseal-

ing of opening **98** pressurized air within pressure chamber **79** rapidly flows through opening **98**, through sealing plate bore **101**, through magazine mounting plate opening **91**, and into launch tube **90** in register with the sealing plate **100** where it propels the projectile **P** from barrel **89**. Operation of this type of release valve is described in more detail in U.S. Pat. No. 4,159,705.

Upon the release of pressurized air from pressure chamber **79** the pressurized air within indexer pressure cylinder **119** is conveyed through pressure tube **120** back into pressure chamber **79**. This release of pressurized air from indexer pressure cylinder **119** causes the indexer piston **121** to be spring biased by coil spring **123** back downward to its low pressure position. The downward movement of piston **121** pivotally retracts the indexing finger **125** from mounting plate groove **95** and positions the finger in register with the following mounting plate groove.

The pump plunger **129** may then be manually drawn back to its initial position to pressurize and fire the gun again. The drawing back of the pump plunger **129** does not create a vacuum within pump cylinder **128** since replenishment air may be drawn through vent **136** into the plunger handle **132**, through the interior of shaft **130**, and through check valve **135** into cylinder **128**. Air between the pump cylinder **128** and the plunger **129** behind seal **134** is expelled from cylinder **128** through vent **141**.

It should be noted that pawl **94** engages notches **93** to prevent rotation of the magazine **75** in a direction opposite to its indexing direction, i.e. to prevent clockwise rotation of the magazine with reference to FIGS. 7 and 8. This prevents the firing of pressurized air into a previously emptied barrel and damage to the indexing finger **125**.

As an alternative, gun **70** may also be constructed without control valve **81**. The need for the control valve is dependent upon the length and interior diameter of pressure tube **78**, i.e. the volume of air contained within the pressure tube. For a pressure tube **78** having a small interior volume the release of air therefrom causes rapid actuation of release valve **80**. Conversely, with a pressure tube **78** containing a large volume of air therein the release of air therefrom may be inadequate to actuate the release valve properly. Thus, with pressure tubes having a large volume therein a control valve **81** is coupled to the release valve **80** to ensure rapid decompression within release valve manifold **105** to actuate the release valve. The gun may also be constructed without the inner launch tube **90** within the barrel **89**. Here, the pressurized air expelled from pressure chamber **79** is directed into barrel **89** behind the projectile. This design however is not preferred as it does not concentrate the burst of pressurized air for optimal efficiency and performance. Lastly, it should be understood that the magazine and indexer of FIGS. 6-9 may also be adapted to a hand held gun of conventional design.

It should be understood that the gun of FIGS. 6-9 may also be adapted to include the two concentric circle arrangement of the opening, as shown in FIGS. 1-5, to increase the dart capacity of the magazine.

With the air gun of this construction a child may aim the gun simply by facing the intended target and manually actuating the hand pump. Because of the elongated, flexible pressure tube **78** the pump may be manipulated substantially independently of and without effecting the air of the launch tube. Thus, the gun is of an unconventional design to interest children yet is capable of being easily aimed and fired. Also, the child may fire several shots sequentially without having to reload between each successive shot.

With reference next to FIGS. 10 and 11, a compressed air gun 159 in another preferred form is shown. Here, the air gun 159 is similar in basic construction to that shown in FIGS. 1-5, except for the internal components for the sequential firing of pressurized air bursts and pneumatic indexing of the magazine, and the magazine grooves 160 are angled rather than being L-shaped. For this reason, only the new, alternative components of the air gun are shown for clarity and conciseness of explanation.

The air gun 159 has a pneumatic firing actuator 161 coupled to the pressure tank through pressure tube 56. Actuator 161 includes an elongated manifold 162 having an inlet opening 163 in fluid communication with pressure tube 56, an outlet opening 164 in fluid communication with a small pressure tank or pressure cell 165, and an open end or firing opening 166 in fluid communication with an elongated recess 167. A piston 168 is mounted for reciprocal movement within actuator manifold 162. Piston 168 has a forward seal 169, a rearward seal 170 and a clear button 171 extending through the air gun housing. The actuator 161 also has a flexible gasket 172 mounted within recess 167 in sealable contact with magazine 18, and a pressure cylinder 173 in fluid communication with pressure cell 165 by a conduit 174. A piston 175, having an elongated piston rod 176, is mounted within the actuator pressure cylinder 173 for reciprocal movement therein between a low pressure, pressurizing position shown in FIG. 10 and a high pressure, firing position shown in FIG. 11. A coil spring 177 mounted about piston rod 176 biases the piston 175 towards its low pressure position. Piston rod 176 is coupled to piston 168 by an over center torsion spring 179. An indexing finger 180, mounted to an end of the piston rod 176, is configured to sequentially engage and ride within each magazine groove 160 for sequential rotation of the magazine.

In use, an operator actuates the pump to pressurize a supply of air by grasping the handle 22 and reciprocating the cylinder rod 21 back and forth within the cylinder 20. With piston 168 in its rearward pressurized air is passed through pressure tube 16 into the pressure tank 15. Manual actuation of the trigger 13 moves the trigger to a position wherein it unpinches pressure tube 56 so as to allow pressurized air within the pressure tank 15 to pass through pressure tube 56 into actuator manifold 162 through inlet opening 163 and between the forward and rearward seals 169 and 170 of piston 168. The pressurized air then passes out of manifold 162 through outlet opening 164 and into pressure cell 165, conduit 174, and pressure cylinder 173. The pressurized air within the pressure cylinder 173 causes piston 175 to move toward its high pressure position against the biasing force of coil spring 177, i.e. the piston 175 is moved from its low pressure position shown in FIG. 10 to its high pressure position shown in FIG. 11.

As shown in FIG. 11, forward movement of the piston 175 causes compression and rotation of torsion spring 179 and the indexing finger 180 to move forward into a magazine groove 160, thereby causing rotation of the magazine 18 and alignment of the opening to change to the inner circle of openings 28". All references herein to forward and rearward is for purposes of clarity in reference to the drawings. Upon reaching the apex of the movement of piston rod 176 the torsion spring 179 reaches a rotated position which causes decompression of the spring thereby forcing piston 168 rearward, as shown in FIG. 11. Rearward movement of piston 168 causes the forward seal 169 to be moved to a positioned between inlet opening 163 and the outlet opening 164. This positioning of the piston 168 isolates manifold inlet opening 163 to prevent escape of pressurized air from

pressure tank 15, i.e. the seals sandwich the inlet opening to prevent the flow of air from the pressure tank. This positioning of the forward seal 169 also allows pressurized air within the pressure cell 165, conduit 174 and pressure cylinder 173 to flow through outlet opening 164 into the manifold and from the manifold through firing opening 166, through sealed recess 167 and into the launch tube 27 through magazine opening 28". Pressurized air within launch tube 27 propels the projectile out of the magazine barrel 26 and through gun barrel 12.

The release of pressurized air from pressure cylinder 173 causes the piston 175 to be spring biased by coil spring 177 back rearward to its low pressure position. The rearward movement of piston 175 retracts the indexing finger 180 from within a mounting plate groove 160 and positions the finger in register with the following mounting plate groove 160. The low pressure positioning of piston 175 causes the torsion spring 179 to bias piston 168 forwards to its initial position with the forward and rearward seals 169 and 170 sandwiching or straddling inlet and outlet openings 163 and 164, as shown in FIG. 10. This repositioning of piston 168 once again causes pressurized air within pressure tank 15 to flow through pressure tube 56 into actuator manifold 162, thereby completing a firing cycle. The firing and indexing cycle just describe may continue in rapid sequence so long as the trigger is maintained in a position allowing the flow of pressurized air through pressure tube 56 and the pressure tank continues to contain a minimal level of pressurized air sufficient to overcome the biasing force of springs 177 and 179, i.e. the release valve is automatically actuated by actuator 161 and the indexing of magazine 18 continues so long as the trigger is pulled open and the pressure tank contains pressurized air above a level to overcome springs 177 and 179. Should the pressure level within pressure tank 15 reach the minimal level the operator simply actuates the manual air pump 14 so as to once again elevate the pressure within the pressure tank.

As described, the gun may be used in a fully automatic manner such that with the trigger maintained in a pulled back, actuated position the gun fires a series of projectiles without stopping between each successive shot, similar to the action of a machine gun. However, should an operator wish to fire a single projectile, one need only to pull the trigger and quickly release it so that pressurized air does not continue to flow into the actuator 161. Operated in such a manner the gun will index the magazine and fire a projectile with each actuation of the trigger, again, so long as the pressure tank contains air pressurized above the minimal level and the trigger is quickly released.

It should be understood that at times rubber seals often stick when stored for a period of time. This sticking may hamper the performance of the actuator. For this reason, the actuator is provided with clear button 171 which may be manually actuated to cause reciprocal movement of the piston in order to unstick the seals.

With reference next to FIGS. 12-15, there is shown a compressed air gun in another preferred embodiment, with like numbers referring to previously described components. Here, the air gun has a combination control valve and indexer 200 which controls the flow of compressed air from the pressure tank 15 to the magazine launch tubes 201 and indexes the magazine 202 with each firing, hereinafter referred collectively as control valve 200.

The control valve 200 has an elongated, cylindrical, external tube or manifold 204, a cylindrical, internal tube 205 mounted within the external tube 204, and a plunger 206

mounted within the internal tube. The external tube **204** has an elongated slot **208**, an air inlet **209** in fluid communication with pressure tube **56**, and an air outlet **210** in fluid communication with magazine launch tubes **201**. The internal tube **205** is configured to move reciprocally within the external tube between a forward position shown in FIG. **13** and a rearward position shown in FIGS. **14–16**. The internal tube **205** and external tube **204** define a first air pressure chamber **212** therebetween, while the internal tube **205** and plunger **206** define a second air pressure chamber **213** therebetween. The internal tube **205** has an air release valve **215**, an O-ring seal **216** for sealing engagement of the internal tube with the external tube, and an L-shaped member **218** extending through slot **208**. L-shaped member **218** has an end flange **219**.

Plunger **206** is mounted within the internal tube **205** for reciprocal movement between a first sealing position abutably sealing air outlet **210** as shown in FIG. **13**, a second sealing position extending from the internal tube yet still sealing air outlet **210** as shown in FIGS. **14** and **15**, and an unsealing position distal from and unsealing air outlet **210** as shown in FIG. **16**. The air release valve **215** has an opening **221**, a plunger **222** mounted within opening **221**, an elongated rod **223**, and a coil spring **224** mounted about elongated rod **223**. The air gun also has a spring biased trigger **227** configured to releasably engage the internal tube L-shaped member **218**.

A coil spring **229** is mounted within internal tube **205** so as to abut plunger **206** and bias the plunger in a direction towards the air outlet **210**. Another coil spring **230** is mounted between the external tube **204** and the internal tube **205** so as to bias the internal tube in a direction towards the air outlet **210**.

The magazine **202** has an annular array of Z-shaped grooves **232** sized and shaped to receive the end flange **219** of the L-shaped member **218**. Each groove **232** has a forward camming surface **233** extending to a forward portion **234** and a rearward camming surface **235** extending to a rearward portion **236**.

In use and with the trigger **227** spring biased to its position engaging the internal tube L-shaped member **218**, the internal tube **205** is initial spring biased to its forward position by compressing spring **230**, as shown in FIG. **13**. This position of the internal tube forces spring **229** to bias plunger **206** to its sealing position. With the internal tube **205** in its forward position, the L-shaped member flange **219** resides within the Z-shaped groove forward portion **234**, as shown in FIG. **21**. It should be understood that the magazine of FIGS. **21** and **22** is illustrated with only one launch tube for clarity of explanation.

As compressed air flows from the pressure tube **56**, extending from the pressure tank **15**, and into the control valve **200** through air inlet **209**, the pressure within the first air pressure chamber **212** increases. Compressed air also passes from the first air pressure chamber, between the plunger **206** and the internal tube, into the second air pressure chamber **213**. The air pressure within the first and second air pressure chambers aid in maintaining the plunger **206** in its sealing position, as the pressure upon the backside of the plunger is greater than ambient air pressure upon the front side of the plunger.

As shown in FIG. **14**, with movement of the trigger **227** to its release position disengaged from the L-shaped member, the compressed air within the first air pressure chamber **212** causes the internal tube **205** to move to its rearward position. This movement of the internal tube

compresses spring **230**. As the internal tube moves rearward the L-shaped member flange **219'** contacts the rearward camming surface **235**, as shown in phantom lines in FIG. **22**. With continued rearward movement of the internal tube, flange **219''** continues into the rearward portion **236** of the Z-shaped groove, as shown in FIG. **22**. The force of the flange upon the rearward camming surface causes the magazine to rotate clockwise approximately half the distance of a complete indexing cycle.

As the internal tube approaches the end of its rearward stroke the release valve spring **224** compresses to a point wherein the force of the spring overcomes the force of the air pressure within the second air pressure chamber **213**. This spring force causes the valve plunger **206** to move forward thereby unseating and allowing the compressed air within the second air pressure chamber **213** to escape rapidly therefrom through opening **221**, as shown in FIG. **15**. This rapid decompression of the second air pressure chamber **213** causes plunger **206** to snap back to its unsealing position, as shown in FIG. **16**. With the plunger in its unsealing position, the compressed air within the first pressure chamber **212** quickly passes through the air outlet **210** and into the launch tube **201**.

The release of the compressed air within the first air pressure chamber **212** causes the internal tube to move forward, through the spring biasing force of coil spring **230**. The forward movement of the internal tube causes the L-shaped member flange **219'''** to contact the forward camming surface **233**, as shown in phantom lines in FIG. **22**, and thus force the remaining indexing rotation of the magazine as the flange **219** once again resides within the forward portion **234**, as shown initially in FIG. **21**.

It should be understood that so long as the trigger is actuated to its disengaged position and so long as there is sufficient air pressure flowing from the pressure tube, the control valve will continue to fire projectiles, as the internal tube and plunger will continue to reciprocate as long as a sufficient amount of compressed air is present to overcome the forces of the springs. Alternatively, the trigger may be pulled and immediately released so that it reengages the L-shaped member after firing a single projectile.

With reference next to FIGS. **17–20**, there is shown the internal components and a portion of the magazine of a compressed air gun in another preferred embodiment, similar to that previously described in reference to FIGS. **12–16**. Here again, the air gun has a combination control valve and indexer **300** which controls the flow of air from the pressure tank **15** to the magazine launch tubes **201** and indexes the magazine **202** with each firing, hereinafter referred collectively as control valve. The control valve **300** has an elongated, cylindrical, external tube or manifold **304**, an internal tube **305** mounted within the external tube **304**, and a plunger **306** mounted within the internal tube. The external tube **304** has an elongated slot **308**, an air inlet **309** in fluid communication with pressure tube **56**, and an air outlet **310** in fluid communication with magazine launch tubes **201**. The internal tube **305** is configured to move reciprocally within the external tube between a forward position, shown in FIG. **17** and a rearward position, shown in FIGS. **18–20**. The internal tube **305** and external tube **304** define an air pressure chamber **312** therebetween. The internal tube **305** has an O-ring seal **316** for sealing engagement of the internal tube with the external tube, and an L-shaped member **318** extending through slot **308**. L-shaped member **318** has an end flange **219**. A coil spring **329** is mounted about the plunger **306** for biased movement of the plunger in a rearward direction.

Plunger 306 is mounted within the internal tube for reciprocal movement between a first sealing position abutably sealing air outlet 310 as shown in FIG. 17, a second sealing position extending from the internal tube yet still sealing air outlet as shown in FIGS. 18 and 19, and an unsealing position distal from and unsealing air outlet as shown in FIG. 20. The air gun also has a spring biased trigger 327 configured to releasably engage the internal tube L-shaped member 318.

A coil spring 330 is mounted about plunger 306 between the forward end of the internal tube and a sealing head 331 of the plunger. Coil spring 330 biases the plunger in a direction towards the air outlet. Another coil spring 328 is mounted between the external tube 304 and the internal tube so as to bias the internal tube in a direction towards the air outlet.

The magazine 202 has an annular array of Z-shaped grooves 232 sized and shaped to receive the end flange 219 of the L-shaped member 318. Each groove 232 has a forward camming surface 233 extending to a forward portion 234 and a rearward camming surface 235 extending to a rearward portion 236.

In use and with the trigger 327 is spring biased to its position engaging the internal tube L-shaped member, the internal tube 305 is initial spring biased to its forward position compressing spring 330. This position of the internal tube forces spring 330 to bias plunger 306 to its sealing position. With the internal tube 305 in its forward position, the L-shaped member flange 219 resides within the Z-shaped groove forward portion 234, as shown in FIG. 21.

As compressed air flows from pressure tube 56 and into the control valve 300 through air inlet 309, the pressure within air pressure chamber 312 increases. This air pressure aids in maintaining the plunger in its sealing position, as the pressure upon the backside of the plunger is greater than ambient air pressure upon the front side of the plunger.

As shown in FIG. 18, with movement of the trigger to its release position disengaging the L-shaped member, the compressed air within the air pressure chamber 312 causes the internal tube 305 to move to its rearward position. This movement of the internal tube compresses springs 328 and 329. As the internal tube moves rearward the L-shaped member flange 219' contacts the rearward camming surface 235 so as to cause the magazine to rotate clockwise approximately half the distance of a complete indexing cycle, as shown in phantom lines in FIG. 22. The flange 219' continues into the rearward portion 236 of the Z-shaped groove.

As the internal tube moves to the end of its rearward stroke the plunger spring 329 compresses to a point wherein the force of spring 329 overcomes the force of the compressed air within the air pressure chamber 312 and upon the plunger sealing head 331. This spring force causes the plunger 306 to move rearwardly to its unsealing position, thereby allowing the compressed air within the air pressure chamber to escape through the air outlet 310, as shown in FIG. 19. The release of the air pressure force upon the plunger allows spring 329 to force plunger 306 quickly rearward to maximize the rapid decompression of the air pressure chamber 312, as shown in FIG. 19.

The release of the compressed air within the air pressure chamber 312 causes the internal tube to move forward, through the spring biasing force of coil spring 328. The forward movement of the internal tube causes the L-shaped member flange 219'' to contact the forward camming surface 233, as shown in phantom lines in FIG. 22, and thus force the remaining indexing rotation of the magazine as the

flange once again resides within the forward portion 234, as shown initially in FIG. 21. Again, the internal tube and plunger may continue to reciprocate as long as the trigger is disengaged and there is sufficient air pressure.

It should be understood that the second air pressure chamber 213 of FIGS. 13–16 performs the same function as spring 329 in FIGS. 17–20, as they both function to snap the plunger rearward upon initial firing.

The gun shown in FIGS. 17–29 may also be adapted to include an internal flange 340, shown in phantom lines, extending from the external tube 305. Flange 340 has an opening 341 therethrough through which plunger 306 extends. Spring 330 abuts flange 340 so that the spring is slightly compressed to force plunger 306 towards its sealing position. As the internal tube 305 moves rearward the spring 330 is compressed further. As air is released from the first air chamber 312, as previously described, spring 330 decompresses so as to force plunger 306 to its sealing position.

It should also be understood that compressed air may be directed into the control valve without the use of a pressure tank 15, as shown in reference to FIGS. 6–9. As such, the control valve may be coupled directly to a pump. Also, the triggering of the control valve, and thus the toy gun, may be accomplished through a valve or regulator mounted between the pressurized air source and the control valve, as shown in the previous embodiments.

While this invention has been described in detail with particular reference to the preferred embodiments thereof, it should be understood that many modifications, additions and deletions, in addition to those expressly recited, may be made thereto without departure from the spirit and scope of invention as set forth in the following claims.

We claim:

1. A compressed air toy gun comprising:

pump means for compressing air;

launch tube means for holding a projectile;

conduit means for conveying compressed air from said pump means to said launch tube means; and

control valve means for controlling the flow of compressed air from said pump means to said launch tube means, said control valve means comprising an external housing having an air inlet in fluid communication with said conduit means and an air outlet in fluid communication with said launch tube means, an internal housing mounted for reciprocal movement within said external housing between a first position and a second position, internal housing biasing means for biasing said internal housing towards said first position, a plunger mounted within said internal housing for reciprocal movement between a sealing position sealing said air outlet and an unsealing position unsealing said air outlet, plunger biasing means for biasing said plunger toward said sealing position, actuation means for actuating the movement of said plunger to said unsealing position, said external housing and said internal housing at least partially defining an air pressure chamber in fluid communication with said air inlet,

whereby compressed air from the pump means is conveyed through the conduit means and through the air inlet into the air pressure chamber, and the compressed air within the air pressure chamber biases the internal housing to its second position whereupon the actuation means causes the movement of the plunger is moved to its unsealing position allowing the compressed air within the air pressure chamber to flow through the air outlet to the launch tube.

15

2. The compressed air toy gun of claim 1 further comprising a pressure tank in fluid communication with said conduit means.

3. The compressed air toy gun of claim 1 wherein said internal housing biasing means comprises a spring.

4. The compressed air toy gun of claim 1 wherein said plunger biasing means comprises a supply of compressed air within said air pressure chamber.

5. The compressed air toy gun of claim 1 wherein said plunger and said internal housing define a second air chamber therebetween, and wherein plunger biasing means comprises a supply of compressed air contained within said second air chamber.

6. The compressed air toy gun 5 wherein said actuation means comprises an air release valve which releases compressed air from said second air chamber.

7. The compressed air toy gun of claim 6 wherein said internal housing biasing means comprises a spring.

8. The compressed air toy gun of claim 1 wherein said actuation means comprises engagement means coupled to said internal housing adapted to engage said plunger as said internal housing is biased to said second position.

9. The compressed air toy gun of claim 4 wherein said actuation means comprises engagement means extending from said internal housing adapted to engage said plunger as said internal housing is biased to said second position.

10. The compressed air toy gun of claim 9 wherein said internal housing biasing means comprises a spring.

11. The compressed air toy gun of claim 1 further comprising triggering means for releasably triggering the movement of said internal housing to said second position.

12. A compressed air toy gun comprising:

pump means for compressing air;

launch tube means for holding a projectile;

conduit means for conveying compressed air from said pump means to said launch tube means; and

control valve means for controlling the flow of compressed air from said pump means to said launch tube means, said control valve means comprising an external tube having an air inlet in fluid communication with said conduit means and an air outlet in fluid communication with said launch tube, an internal tube mounted within said external tube for reciprocal movement between a first position adjacent said air outlet and a second position distal said air outlet, a plunger mounted within said internal tube for reciprocal movement between a sealing position sealing said air outlet and an unsealing position unsealing said air outlet, spring biasing means for biasing said internal tube towards said first position, said external tube and said internal tube defining a first pressure chamber for releasably storing compressed air, said internal tube and said plunger defining a second pressure chamber for releasably storing compressed air, an air release valve in fluid communication with said second pressure chamber to controllably releasing the compressed air within said second pressure chamber,

whereby compressed air from the pump means is conveyed through the conduit means into the first pressure chamber and between the internal tube and the plunger to the second pressure chamber, the compressed air within the first pressure chamber biases the internal tube to its second position while the compressed air within the second pressure chamber maintains the plunger in its sealing position, and the actuation of the air release valve causes the compressed air within the

16

second pressure chamber to be released thereby causing the plunger to move to its unsealing position thereby allowing the compressed air within the first pressure chamber to flow through the air outlet and into the launch tubes.

13. The compressed air toy gun of claim 12 further comprising a pressure tank in fluid communication with said conduit means.

14. The compressed air toy gun of claim 12 further comprising spring means for biasing said plunger towards said sealing position.

15. The compressed air toy gun of claim 12 further comprising triggering means for releasably triggering the movement of said internal tube to said second position.

16. The compressed air toy gun of claim 12 wherein said air release valve is actuated to release the compressed air within said second air pressure chamber upon said internal tube reaching said second position.

17. A compressed air toy gun comprising:

pump means for compressing air;

launch tube means for holding a projectile;

conduit means for conveying compressed air from said pump means to said launch tube means; and

control valve means for controlling the flow of compressed air from said pump means to said launch tube means, said control valve means comprising an external tube having an air inlet in fluid communication with said conduit means and an air outlet in fluid communication with said launch tube, an internal tube mounted within said external tube for reciprocal movement between a first position adjacent said air outlet and a second position distal said air outlet, a plunger mounted within said internal tube for reciprocal movement between a sealing position sealing said air outlet and an unsealing position unsealing said air outlet, spring biasing means for biasing said internal tube towards said first position, said external tube and said internal tube defining a pressure chamber for releasably storing compressed air, said internal tube having a portion sized and shaped to engage said plunger,

whereby compressed air from the pump means is conveyed through the conduit means into the pressure chamber, the compressed air within the pressure chamber biases the plunger to its sealing position while biasably moving the internal tube to its second position, the movement of the internal tube to its second position causes the plunger to move to its unsealing position thereby allowing the compressed air within the pressure chamber to flow through the air outlet and into the launch tubes.

18. The compressed air toy gun of claim 17 further comprising a pressure tank in fluid communication with said conduit means.

19. The compressed air toy gun of claim 17 further comprising a spring biasing means for biasing said plunger to said sealing position.

20. The compressed air toy gun of claim 17 further comprising triggering means for releasably triggering the movement of said internal tube to said second position.

21. A compressed air toy gun comprising:

pump means for compressing air;

launch tube means for holding a projectile;

conduit means for conveying compressed air from said pump means to said launch tube means; and

control valve means for controlling the flow of compressed air from said pump means to said launch tube

17

means, said control valve means comprising an external tube having an air inlet in fluid communication with said conduit means and an air outlet in fluid communication with said launch tube means, a plunger mounted within said external tube for reciprocal movement between a sealing position sealing said air outlet and an unsealing position unsealing said air outlet, and pressure sensitive actuation means for controlled continuous, reciprocal actuation of said plunger so long as compressed air above a preselected pressure level flows from said conduit means to said control valve means,

whereby compressed air from the pump means is conveyed through the conduit means and through the air inlet to the control valve which continuously reciprocates the plunger for firing a sequence of projectiles.

22. The compressed air toy gun of claim **21** wherein said pressure sensitive actuation means comprises an internal tube mounted for reciprocal movement between a first position and a second position, internal tube biasing means for biasing said internal tube towards said first position.

23. The compressed air toy gun of claim **22** wherein said pressure sensitive actuation means further comprises plunger biasing means for biasing said plunger toward said sealing position, and wherein said external tube and said internal tube at least partially defining an air pressure chamber in fluid communication with said air inlet, whereby compressed air within the pressure chamber moves the internal tube to its second position.

18

24. The compressed air toy gun of claim **21** further comprising a pressure tank in fluid communication with said conduit means.

25. The compressed air toy gun of claim **21** wherein said internal tube biasing means comprises a spring.

26. The compressed air toy gun of claim **23** wherein said plunger biasing means comprises a supply of compressed air within said air pressure chamber.

27. The compressed air toy gun of claim **23** wherein said plunger and said internal tube define a second air chamber therebetween, and wherein plunger biasing means comprises a supply of compressed air contained within said second air chamber.

28. The compressed air toy gun **27** wherein said pressure sensitive actuation means further comprises an air release valve which releases compressed air from said second air chamber.

29. The compressed air toy gun of claim **22** wherein said actuation means comprises engagement means extending from said internal tube adapted to engage said plunger as said internal tube is biased to said second position.

30. The compressed air toy gun of claim **22** wherein said internal tube biasing means comprises a spring.

31. The compressed air toy gun of claim **21** further comprising triggering means for releasably triggering the movement of said pressure sensitive actuation means.

* * * * *