



US 20170168176A1

(19) **United States**(12) **Patent Application Publication**
Tenghamn(10) **Pub. No.: US 2017/0168176 A1**(43) **Pub. Date: Jun. 15, 2017**(54) **LOW FREQUENCY MARINE ACOUSTIC
VIBRATOR****Publication Classification**

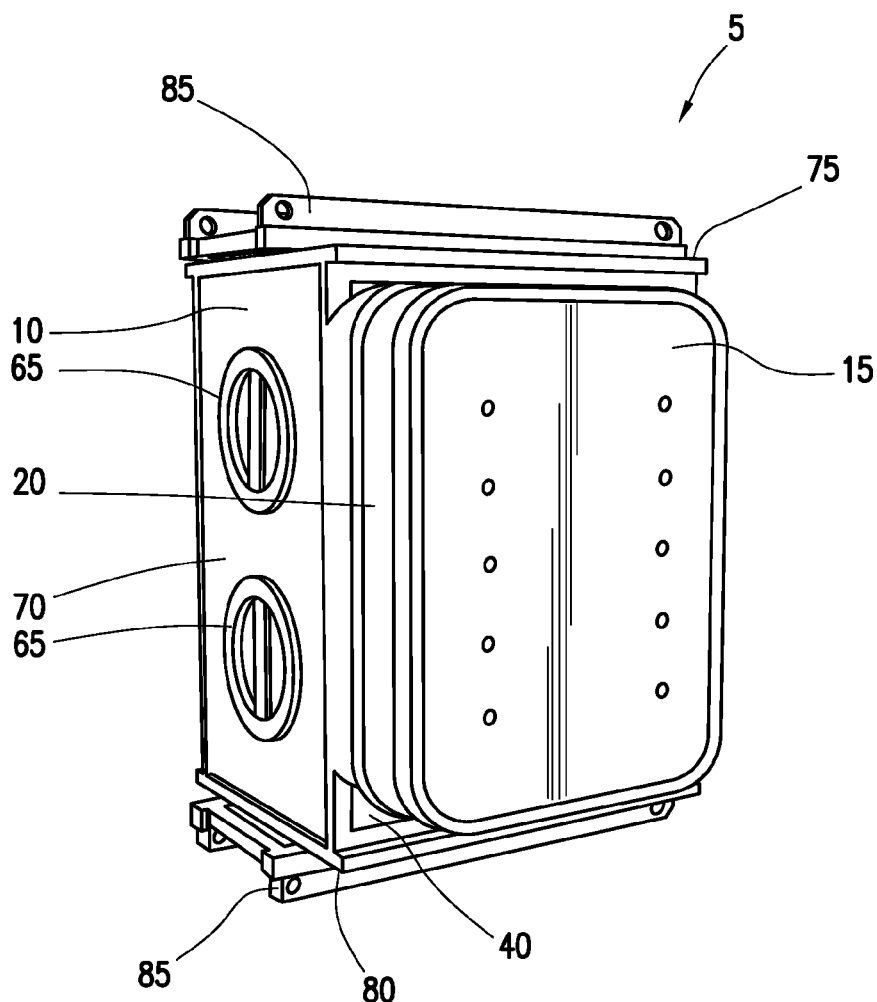
- (51) **Int. Cl.**
G01V 1/137 (2006.01)
G01V 1/38 (2006.01)
- (52) **U.S. Cl.**
CPC **G01V 1/137** (2013.01); **G01V 1/3808**
(2013.01)

(71) Applicant: **PGS Geophysical AS, Oslo (NO)**(72) Inventor: **Stig Rune Lennart Tenghamn,**
Houston, TX (US)(73) Assignee: **PGS Geophysical AS, Oslo (NO)**(21) Appl. No.: **15/443,757**(22) Filed: **Feb. 27, 2017****Related U.S. Application Data**

- (63) Continuation of application No. 14/284,847, filed on May 22, 2014, now Pat. No. 9,618,637.
- (60) Provisional application No. 61/880,561, filed on Sep. 20, 2013.

(57) **ABSTRACT**

Embodiments relate to marine acoustic vibrators that incorporate one or more piston plates that act on the surrounding water to produce acoustic energy. An example marine acoustic vibratory may comprise: a containment housing; a piston plate; a fixture coupled to the containment housing; a spring element coupled to the piston plate and the fixture; and a driver coupled to the piston plate and the fixture and configured to move the piston plate back and forth.



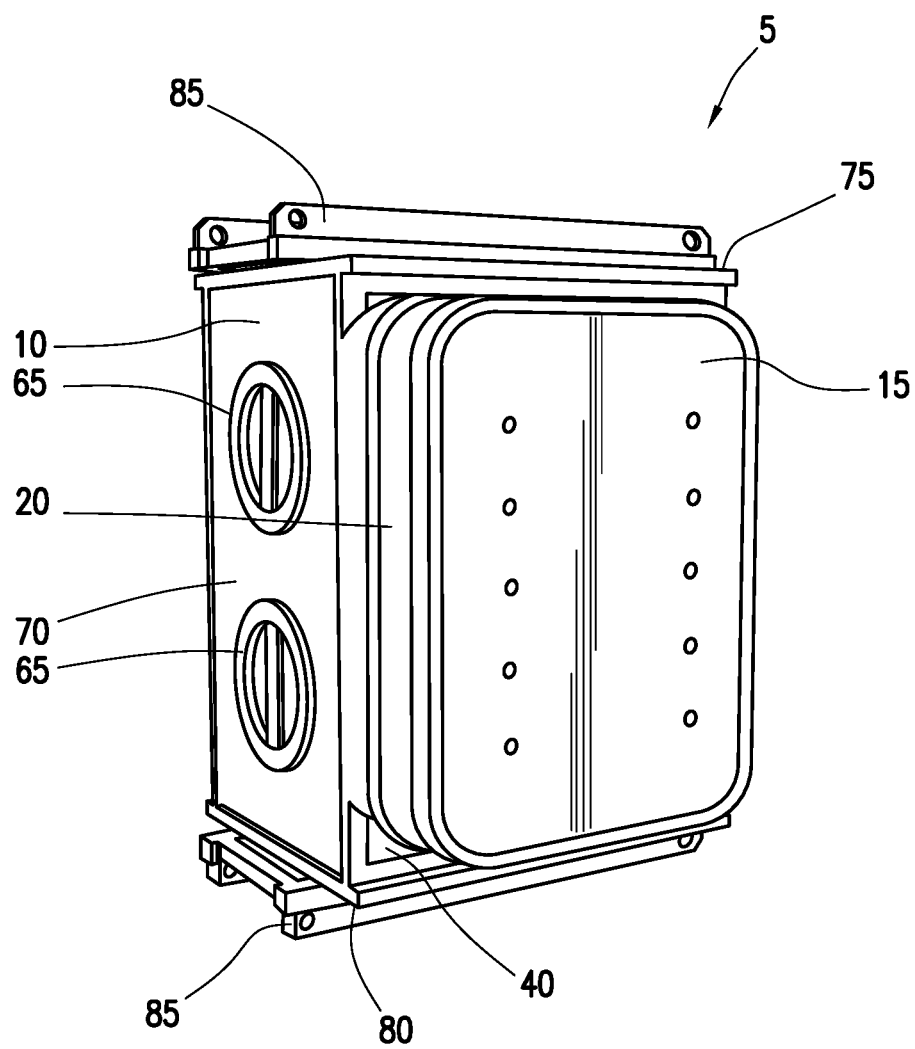


FIG. 1

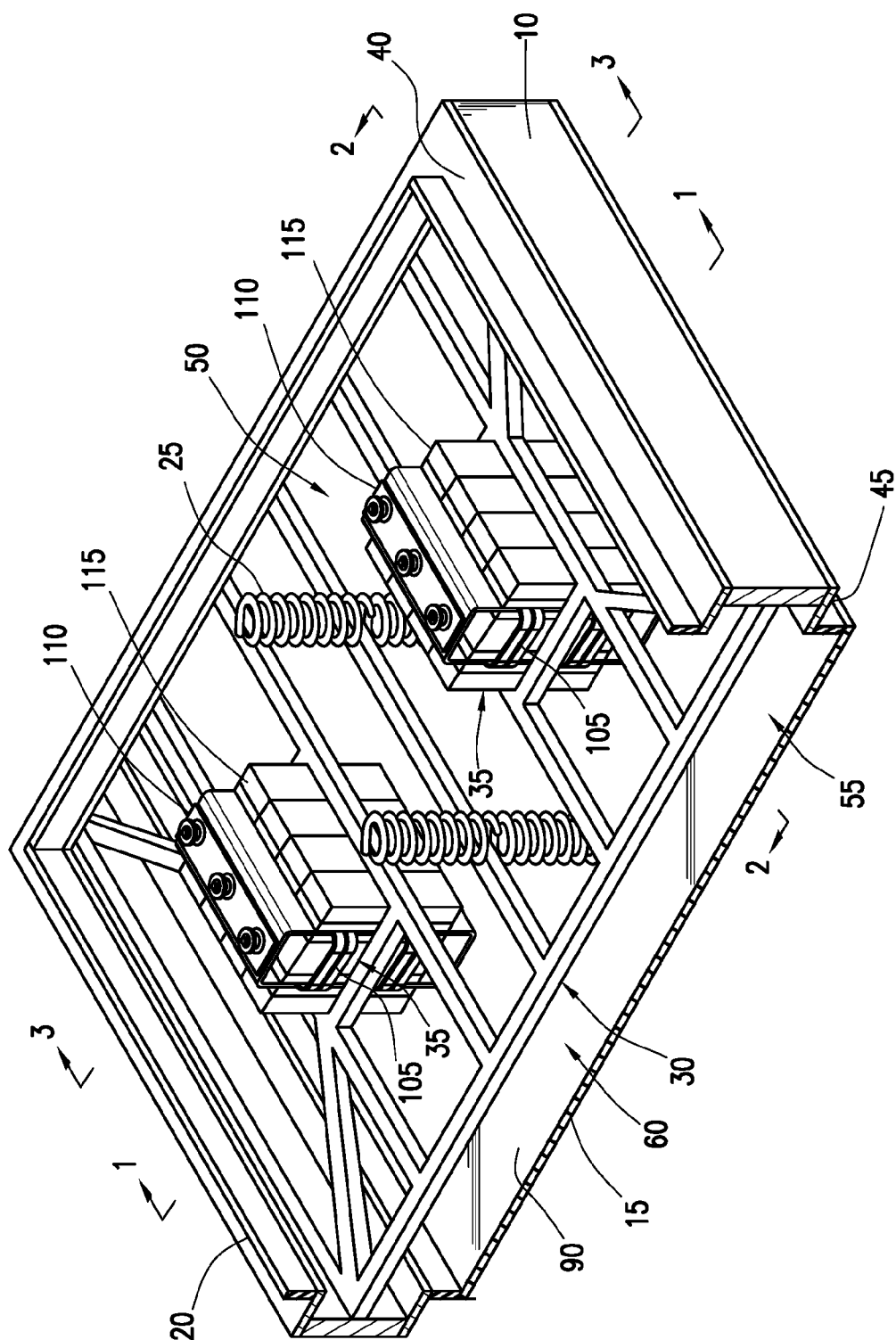


FIG. 2

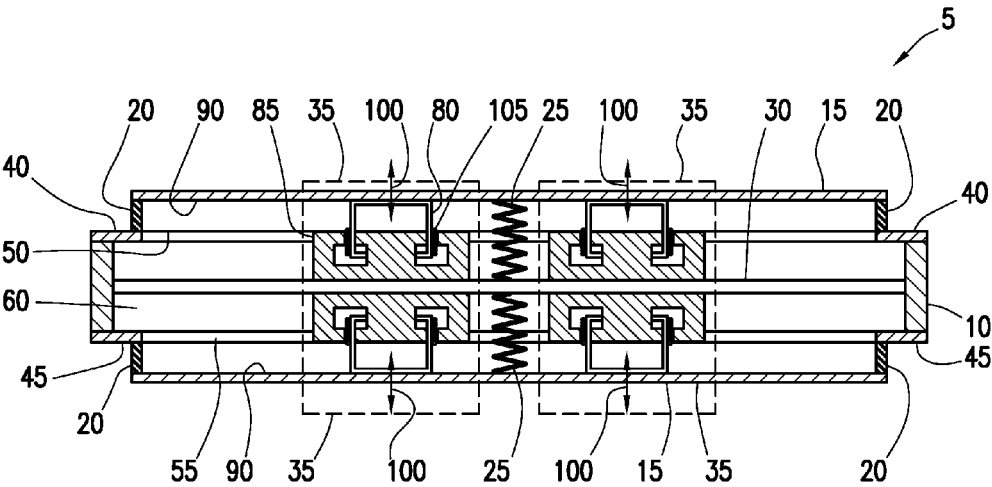


FIG. 3

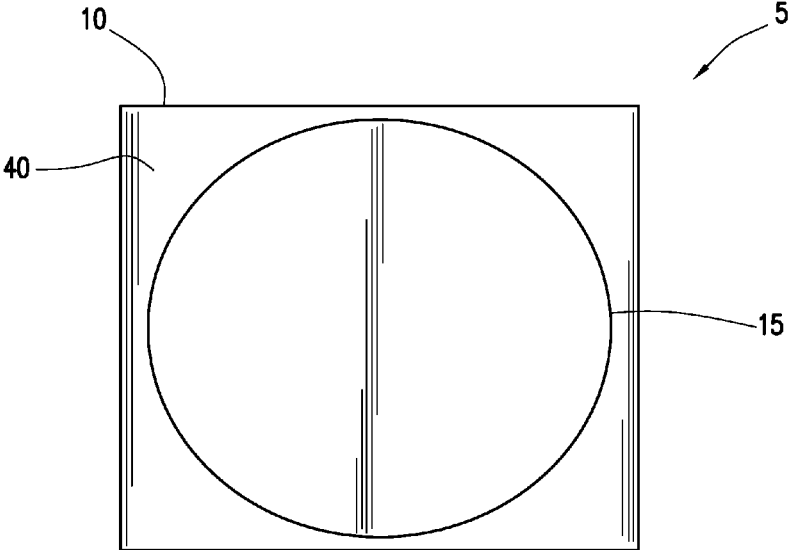


FIG. 5

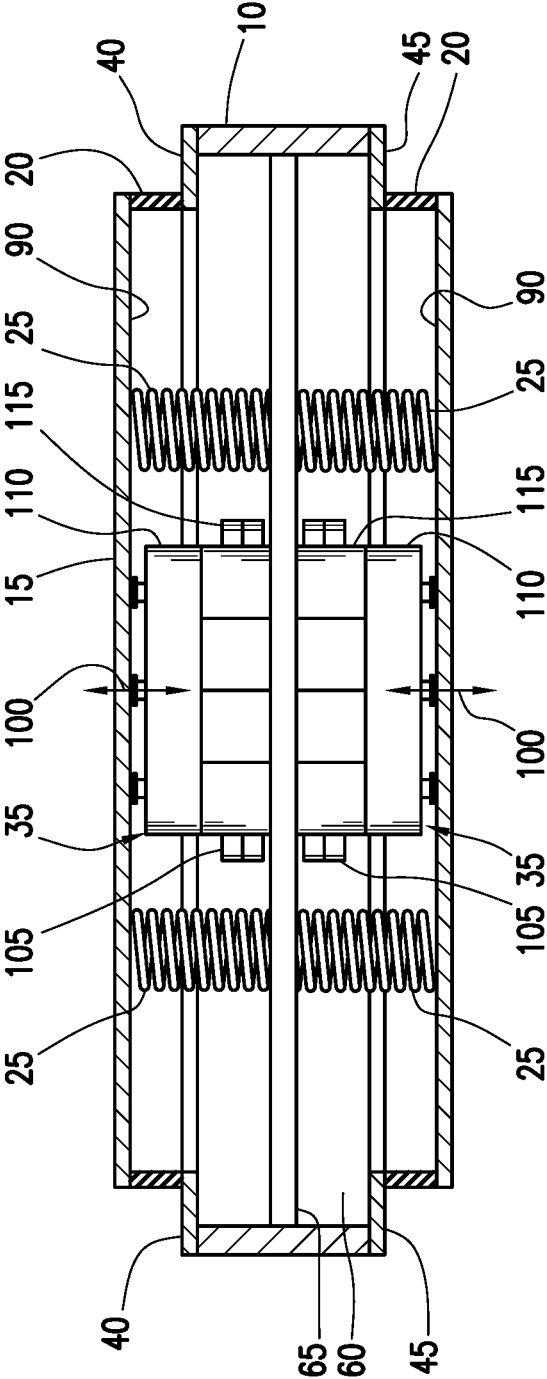


FIG. 4

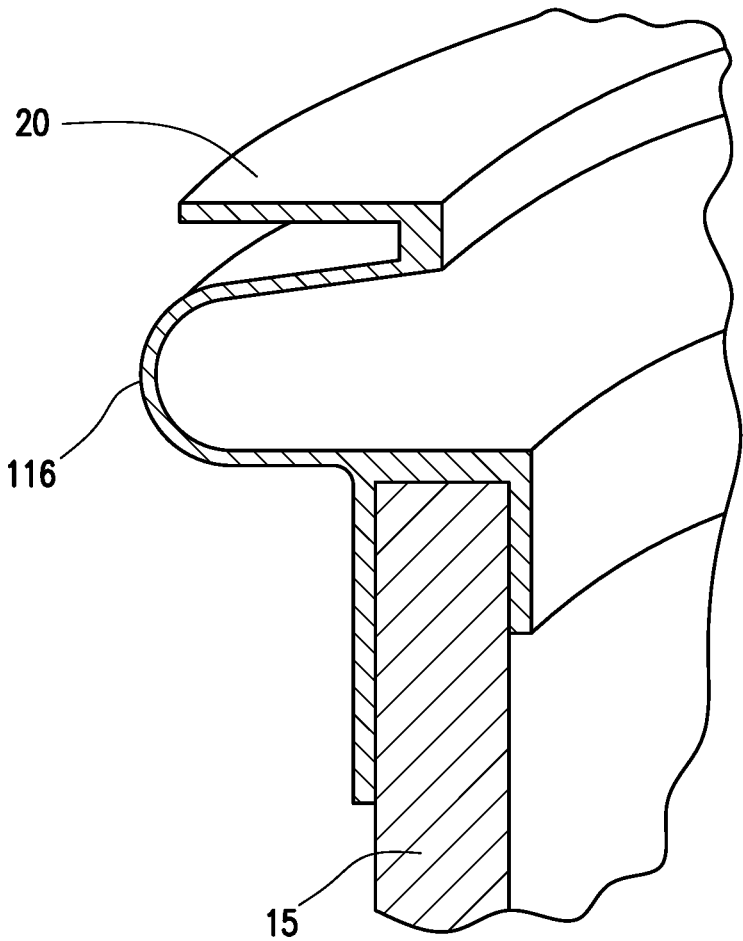


FIG. 6

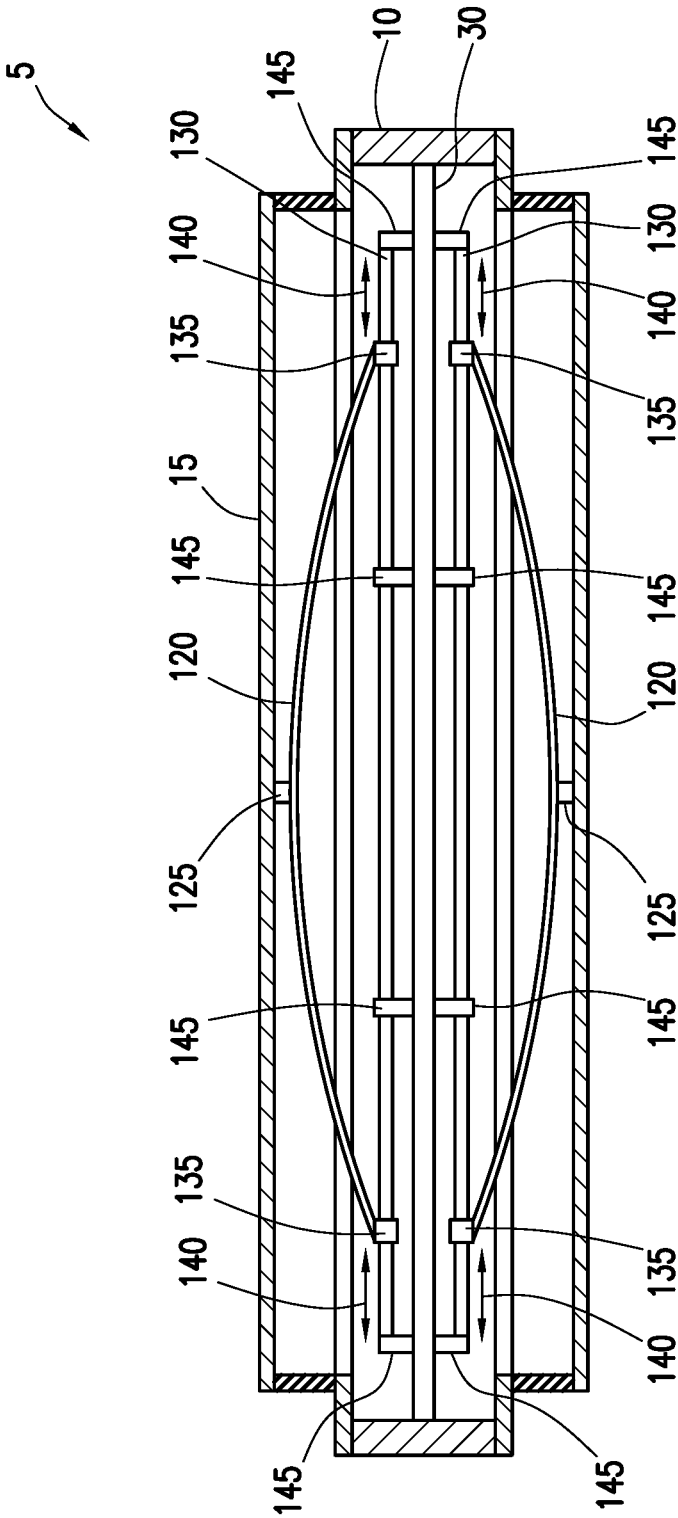


FIG. 7

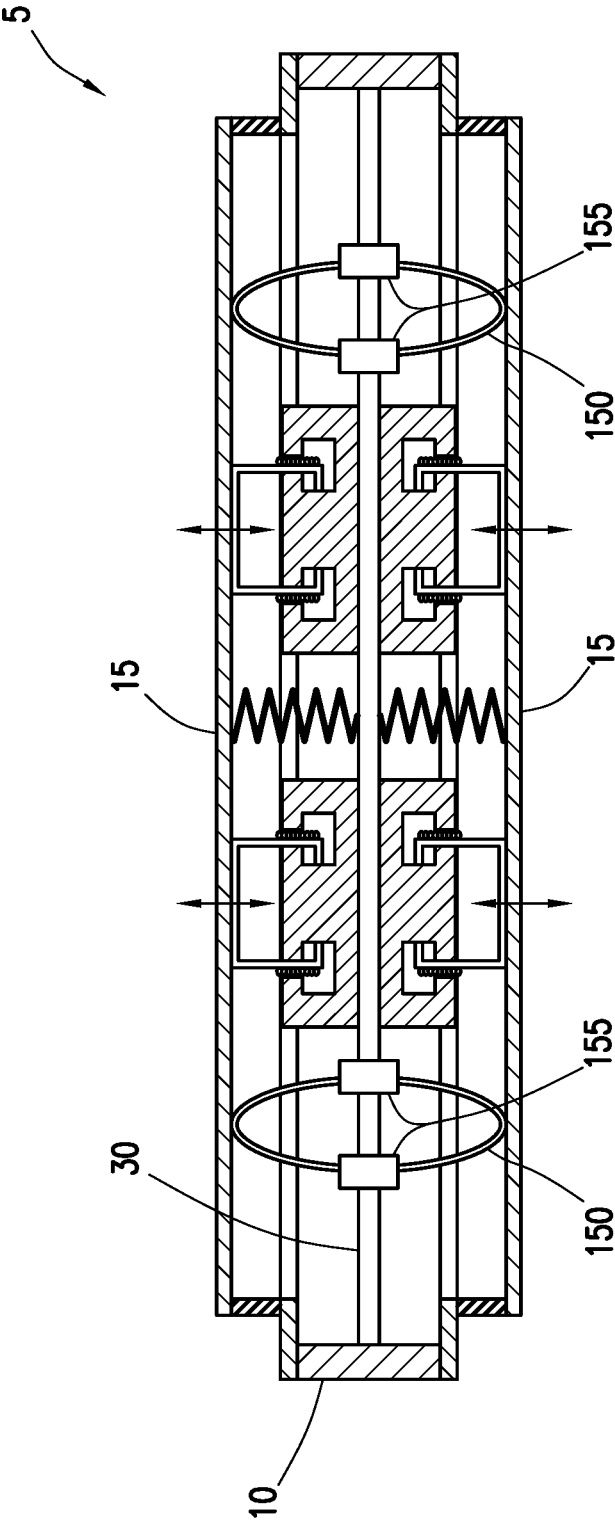
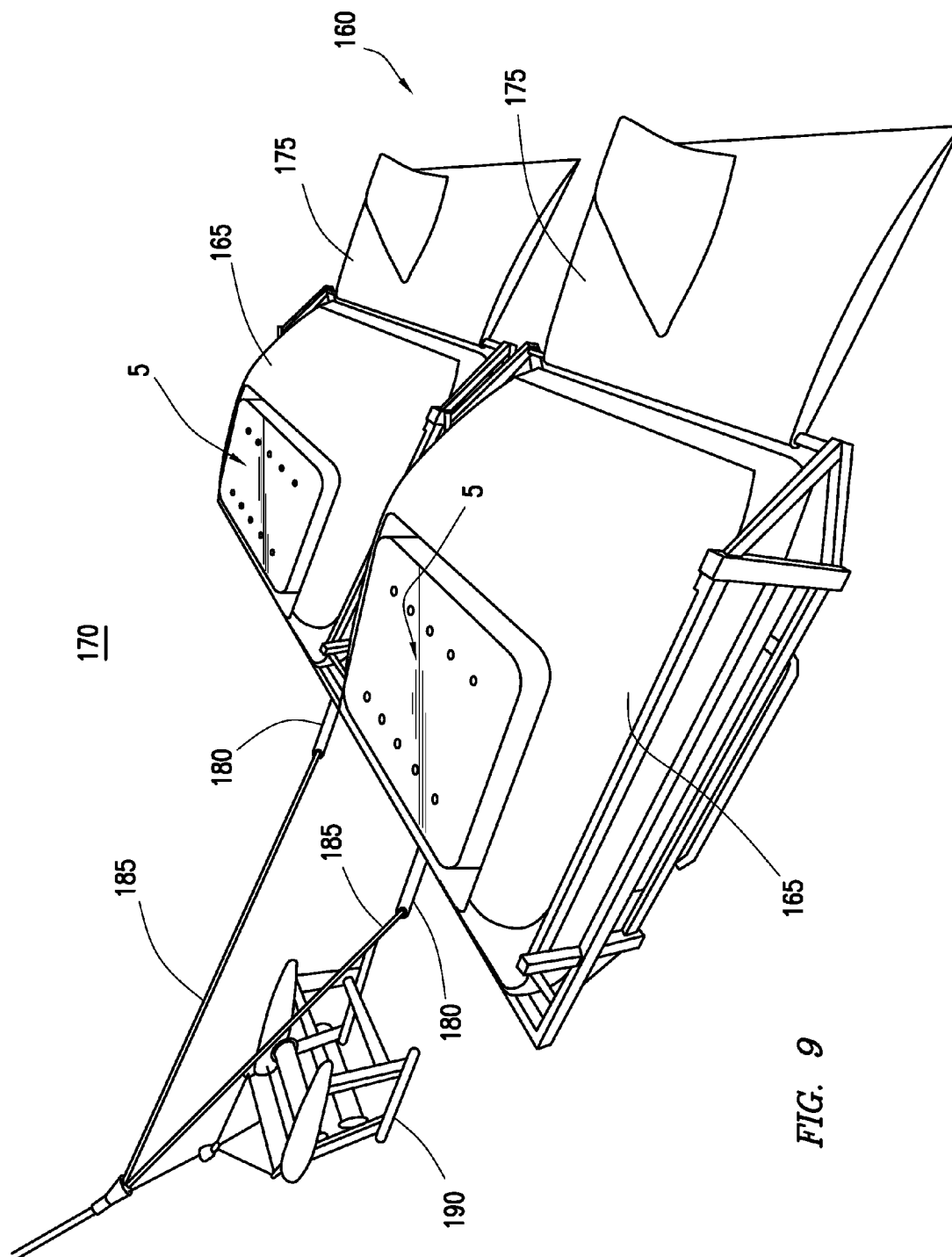


FIG. 8



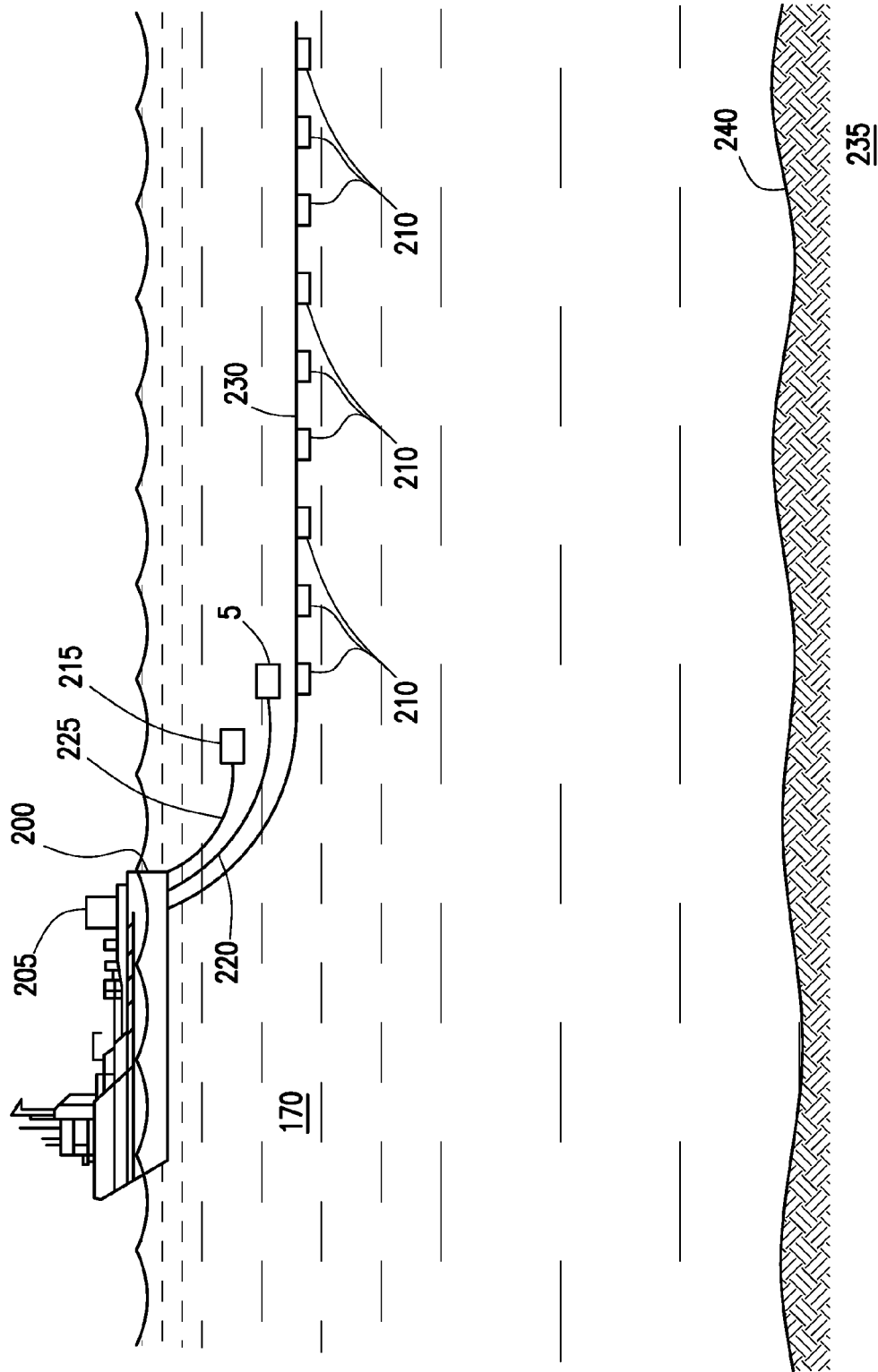


FIG. 10

LOW FREQUENCY MARINE ACOUSTIC VIBRATOR

CROSS-REFERENCE TO RELATED APPLICATIONS

[0001] The present application is a continuation of U.S. patent application Ser. No. 14/284,847, filed on May 22, 2014, which claims priority to U.S. Provisional Application No. 61/880,561, filed on Sep. 20, 2013, the entire disclosures of which are incorporated herein by reference.

BACKGROUND

[0002] Embodiments relate generally to marine seismic surveying. More particularly, embodiments relate to marine acoustic vibrators that incorporate one or more piston plates that act on the surrounding water to produce acoustic energy.

[0003] Sound sources, including marine acoustic vibrators, are generally devices that generate acoustic energy. One use of sound sources is in marine seismic surveying in which the sound sources may be employed to generate acoustic energy that travels downwardly through water and into subsurface rock. After interacting with the subsurface rock, e.g., at boundaries between different subsurface layers, some of the acoustic energy may be returned toward the water surface and detected by specialized sensors in the water, typically either on the water bottom or towed on one or more streamers. The detected energy may be used to infer certain properties of the subsurface rock, such as structure, mineral composition and fluid content, thereby providing information useful in the recovery of hydrocarbons.

[0004] Most of the sound sources employed today in marine seismic surveying are of the impulsive type, in which efforts are made to generate as much energy as possible during as short a time span as possible. The most commonly used of these impulsive-type sources are air guns that typically utilize compressed air to generate a sound wave. Other examples of impulsive-type sources include explosives and weight-drop impulse sources. The frequency content of these impulsive-type sound sources typically in use today is controllable only to a small degree. As a result, different sound sources may be selected for the generation of different frequency ranges based on the surveying needs. Notably, these impulsive-type sources also have limited acoustic energy output in the very low frequency band from 1-10 Hz. However, it is well known that as sound waves travel through water and through subsurface geological structures, higher frequency sound waves may be attenuated more rapidly than lower frequency sound waves, and consequently, lower frequency sound waves can be transmitted over longer distances through water and geological structures than higher frequency sound waves. Thus, efforts have been undertaken to develop sound sources that can operate at low frequencies.

BRIEF DESCRIPTION OF THE DRAWINGS

[0005] These drawings illustrate certain aspects of some of the embodiments of the present invention and should not be used to limit or define the invention.

[0006] FIG. 1 illustrates one embodiment of a marine acoustic vibrator.

[0007] FIG. 2 illustrates a partial cross-sectional view of the marine acoustic vibrator of FIG. 1.

[0008] FIG. 3 illustrates a cross-sectional view of the marine acoustic vibrator of FIGS. 1 and 2 taken along line 1-1 of FIG. 2.

[0009] FIG. 4 illustrates a cross-sectional view of the marine acoustic vibrator of FIGS. 1 and 2 taken along line 2-2 of FIG. 2.

[0010] FIG. 5 illustrates another embodiment of a marine acoustic vibrator.

[0011] FIG. 6 illustrates one embodiment of a rubber seal used with a marine acoustic vibrator.

[0012] FIG. 7 illustrates a cross-sectional view of an embodiment of a marine acoustic vibrator with an alternative embodiment of a mechanical spring element taken along line 3-3 of FIG. 2.

[0013] FIG. 8 illustrates another embodiment of a marine acoustic vibrator.

[0014] FIG. 9 illustrates one embodiment of an array of marine acoustic vibrators.

[0015] FIG. 10 illustrates one embodiment of a marine seismic survey system using a marine acoustic vibrator.

DETAILED DESCRIPTION

[0016] It is to be understood that the present disclosure is not limited to particular devices or methods, which may, of course, vary. It is also to be understood that the terminology used herein is for the purpose of describing particular embodiments only, and is not intended to be limiting. All numbers and ranges disclosed herein may vary by some amount. Whenever a numerical range with a lower limit and an upper limit is disclosed, any number and any included range falling within the range are specifically disclosed. Although individual embodiments are discussed herein, the invention covers all combinations of all those embodiments. As used herein, the singular forms "a", "an", and "the" include singular and plural referents unless the content clearly dictates otherwise. Furthermore, the word "may" is used throughout this application in a permissive sense (i.e., having the potential to, being able to), not in a mandatory sense (i.e., must). The term "include," and derivations thereof, mean "including, but not limited to." The term "coupled" means directly or indirectly connected. If there is any conflict in the usages of a word or term in this specification and one or more patent or other documents that may be incorporated herein by reference, the definitions that are consistent with this specification should be adopted for the purposes of understanding this invention.

[0017] Embodiments relate generally to marine acoustic vibrators that incorporate one or more piston plates that may act on the surrounding water to produce acoustic energy. The marine acoustic vibrators may further comprise one or more drivers coupled to the piston plates to cause the piston plates to move back and forth. The marine acoustic vibrators may also include one or more springs coupled between the piston plates and a fixture in the vibrators. Advantageously, the marine acoustic vibrators may display a low resonance frequency in the seismic frequency range of interest. In particular embodiments, the marine acoustic vibrator may display at least one resonance frequency (when submerged in water at a depth of from about 0 meters to about 300 meters) in the seismic frequency range of interest between about 1 Hz to about 200 Hz. In alternative embodiments, the marine acoustic vibrator may display at least one resonance frequency (when submerged in water at a depth of from about 0 meters to about 300 meters) between about 0.1 Hz

and about 100 Hz and, alternatively, between about 0.1 Hz and about 10 Hz. In some embodiment, the marine acoustic vibrator may display at least two resonance frequencies of about 10 Hz or lower (when submerged in water at a depth of from about 0 meters to about 300 meters).

[0018] An embodiment may provide a marine acoustic vibrator comprising: a containment housing; a piston plate; a fixture coupled to the containment housing; a spring element coupled to the piston plate and the fixture; and a driver coupled to the piston plate and the fixture and configured to move the piston plate back and forth.

[0019] Another embodiment may provide a marine acoustic vibrator comprising a containment housing and a fixture coupled to the containment housing. The marine acoustic vibrator may further comprise a first piston plate coupled to the housing. The marine acoustic vibrator may further comprise a first driver disposed within the marine acoustic vibrator, the first driver being coupled to the fixture and the first piston plate, wherein the first driver is configured to move the first piston plate back and forth. The marine acoustic vibrator may further comprise a first pair of spring elements coupled to the first piston plate and the fixture, wherein the first pair of spring elements are positioned on opposite sides of the first driver from one another. The marine acoustic vibrator may further comprise a second piston plate coupled to the housing opposite the first piston plate. The marine acoustic vibrator may further comprise a second driver disposed within the marine acoustic vibrator, the second driver being coupled to the fixture and the second piston plate, wherein the second driver is configured to move the second piston plate back and forth. The marine acoustic vibrator may further comprise a second pair of spring elements coupled to the second piston plate and the fixture, wherein the second pair of spring elements are positioned on opposite sides of the second driver from one another.

[0020] Yet another embodiment may provide a method comprising: towing a marine acoustic vibrator in a body of water in conjunction with a marine seismic survey; and triggering the marine acoustic vibrator to cause one or more piston plates in the marine acoustic vibrator to move back and forth wherein one or more spring elements exert a biasing force against the one or more piston plates, the spring elements being coupled to the one or more piston plates and a fixture in the marine acoustic vibrator.

[0021] In order to appreciate the exemplary marine acoustic vibrators discussed herein, a discussion of how such marine acoustic vibrators may be able to efficiently transmit acoustic energy is initially presented. As would be understood by one of ordinary skill in the art, the total impedance that may be experienced by a marine acoustic vibrator may be expressed as follows:

$$Z_r = R_r + jX_r \quad (\text{Eq. 1})$$

where Z_r is total impedance, R_r is radiation impedance, and X_r is reactive impedance.

[0022] In an analysis of the energy transfer of a marine acoustic vibrator, the system may be approximated as a baffled piston. In the expression of the total impedance that will be experienced, the radiation impedance R_r of a baffled piston may be:

$$R_r = \pi a^2 \rho_o c R_1(x) \quad (\text{Eq. 2})$$

and the reactive impedance may be:

$$X_r = \pi a^2 \rho_o c X_1(x) \quad (\text{Eq. 3})$$

where

$$x = 2ka = (4\pi a/\lambda) = (2\omega a/c) \quad (\text{Eq. 4})$$

and where

$$R_1(x) = 1 - (2/x)J_1(x) \text{ and} \quad (\text{Eq. 5})$$

$$X_1(x) = \left(\frac{4}{\pi}\right) \int_0^{\pi/2} \sin(x \cos \alpha) \sin^2 \alpha d\alpha \quad (\text{Eq. 6})$$

where ρ_o =density of water, ω =radial frequency, k =wave number, a =radius of piston, c =sound velocity, λ =wave length, and J_1 =Bessel function of the first order.

[0023] Using the Taylor series expansion on the above equations yields the following:

$$R_1(x) = \frac{x^2}{2^2 1! 2!} - \frac{x^4}{2^4 2! 3!} + \dots \quad (\text{Eq. 7})$$

$$X_1(x) = \frac{4}{\pi} \left(\frac{x}{3} - \frac{x^3}{3^2 5} + \frac{x^5}{3^2 5^2 7} - \dots \right) \quad (\text{Eq. 8})$$

[0024] For low frequencies, when $x=2ka$ is much smaller than 1, the real and imaginary part of the total impedance expression may be approximated with the first term of the Taylor expression. The expressions for low frequencies, when the wave length is much larger than the radius of the piston becomes:

$$R_1(x) \rightarrow (1/2)(ka)^2 \quad (\text{Eq. 9})$$

$$X_1(x) \rightarrow (8ka)/(3\pi) \quad (\text{Eq. 10})$$

[0025] It follows that, for low frequencies, R will be a small number compared to X , which suggests a very low efficiency signal generation. However, embodiments may introduce a resonance in the lower end of the frequency spectrum so that low frequency acoustic energy may be generated more efficiently. By way of example, embodiments of the marine acoustic vibrators may display at least one resonance frequency within the seismic frequency range of about 1 Hz to about 10 Hz. At resonance, the imaginary (reactive) part of the impedance is cancelled, and the marine acoustic vibrator may be able to efficiently transmit acoustic energy into the body of water.

[0026] Turning now to FIGS. 1-4, an embodiment of a marine acoustic vibrator 5 will be described. FIG. 1 is a perspective view of an embodiment of the marine acoustic vibrator 5. FIG. 2 is a partial cross-sectional view in perspective of the marine acoustic vibrator 5 of FIG. 1 with one of the piston plates 15 removed. FIG. 3 is a cross-sectional view of the marine acoustic vibrator 5 of FIGS. 1 and 2 taken along line 1-1. FIG. 4 is a cross-sectional view of the marine acoustic vibrator 5 of FIGS. 1 and 2 taken along line 2-2.

[0027] In the illustrated embodiment, the marine acoustic vibrator 5 includes a containment housing 10. Piston plates 15 may be flexibly coupled to the containment housing 10, for example, by way of rubber seals 20. As best seen on FIGS. 3 and 4, the piston plates 15 may each have spring elements 25 attached to them. The spring elements 25 may be disposed between the piston plates 15 and a fixture 30. One or more drivers 35 may be coupled to the piston plates 15 and the fixture 30 to cause the piston plates 15 to move

back and forth. This motion of the piston plates 15 takes advantage of the flexibility of the rubber seals 20. As would be understood by one of ordinary skill in the art with the benefit of this disclosure, the rubber seals 20 do not need to be made of rubber, but rather could be made from any material that allows a flexible coupling of the piston plates 15 to the containment housing 10 as further discussed below.

[0028] The containment housing 10 may have a first surface 40 and a second surface 45, which may be opposing one another. As best seen on FIGS. 2-4, windows or openings 50 and 55 may be formed respectively in the first surface 40 and the second surface 45. While not illustrated, embodiments may include windows or openings 50, 55 that are larger or smaller than the piston plates 15. The containment housing 10 together with the piston plates 15 may at least partially define an interior volume 60 of the marine acoustic vibrator 5. In some embodiments, the spring elements 25 and/or drivers 35 may be at least partially disposed in the interior volume 60 of the marine acoustic vibrator 5. By way of example, a portion of the spring elements 25 and/or drivers 35 may be disposed outside the interior volume 60 of the marine acoustic vibrator. In alternative embodiments, the spring elements 25 and/or drivers 35 may be entirely disposed within the interior volume 60. While not illustrated, the spring elements 25 may be disposed outside the interior volume 60 so long as the spring elements 25 are coupled to the fixture 30. In some embodiments, the marine acoustic vibrator 5 may be pressure compensated such that the pressure of the interior volume 60 may be kept the same as the external pressure (i.e., the pressure on the side of the piston plates 15 opposite that of the interior volume 60), thus enabling operation at increased depth, for example, up to about 300 meters or more. The containment housing 10 together with the piston plates 15 and the rubber seals 20 may form a waterproof housing for the other components of the marine acoustic vibrator 5, such as the spring elements 25 and drivers 35. The containment housing 10 may be constructed from any suitable material, including, without limitation, steel (e.g., stainless steel), aluminum, a copper alloy, glass-fiber reinforced plastic (e.g., glass-fiber reinforced epoxy), carbon fiber reinforced plastic, and combinations thereof. Similarly, the containment housing 10 as best seen in FIGS. 1 and 2 may have the general shape of a rectangular box. It should be understood that other configurations of the containment housing 10 may be suitable, including those having the general shape of a square box or other suitable shapes.

[0029] In some embodiments, the containment housing 10 may further include optional caps 65, which may be disposed in a lateral side 70 of the containment housing 10, as shown on FIG. 1. In particular embodiments, one or more of the caps 65 may be removable. By way of example, the caps 65 may facilitate attachment of a device, such as a compliance chamber, to the containment housing 10. As further illustrated by FIG. 1, the containment housing 10 may include first and second ends 75, 80 to which brackets 85 may be separately mounted. The brackets 85 may be used for hoisting marine acoustic vibrator 5, for example when deploying the marine acoustic vibrator 5 in the water. By way of example, the brackets 85 may facilitate attachment of the marine acoustic vibrator 5 to tow lines, a submersible vessel (e.g., submersible vessel 165 on FIG. 9), or other

suitable device or mechanism used in conjunction with towing the marine acoustic vibrator 5 through a body of water.

[0030] The piston plates 15 may typically be constructed of a material that will not bend or flex when in use. By way of example, the piston plates 15 may comprise, without limitation, steel (e.g., stainless steel), aluminum, a copper alloy, glass-fiber reinforced plastic (e.g., glass-fiber reinforced epoxy), carbon fiber reinforced plastic, and combinations thereof. In some embodiments, the piston plates 15 may be substantially rectangular in shape. By way of example, the piston plate 15 shown on FIG. 1 is rectangular in shape except with rounded corners. In some embodiments, the piston plates 15 may be in the form of flat, circular disks (as best seen in FIG. 5). By way of example, the piston plates 15 may each be a flat, circular disk having substantially uniform thickness. However, other configurations, including both axially-symmetric and not, of the piston plates 15 may be suitable for particular applications. By way of example, the piston plates 15 may be square, elliptical, or other suitable shape for providing the desired acoustic energy. In alternative embodiments, the piston plates 15 may be curved, either convexly protruding into interior volume 60, or concavely extending interior volume 60. In general, the piston plates 15 have a thickness that provides stiffness and also withstands expected pressures. As will be appreciated by those of ordinary skill in the art with the benefit of this disclosure, the plate thickness may vary based on the material of construction, among other factors. As will be discussed in more detail below, the mass load of the piston plates 15 and the spring constant of the spring elements 25 may be selected (i.e., tuned) in a manner to produce a first resonance frequency within the seismic frequency range of interest when the marine acoustic vibrator 5 is submerged in water at a depth of from about 0 meters to about 300 meters. While a single one of the piston plates 15 is illustrated on either side of the fixture 30, embodiments may include multiple piston plates 15 on either side of the fixture 30. Moreover, embodiments may include piston plates 15 that are smaller in size with respect to the containment housing 10 as compared to those illustrated on FIGS. 1-4.

[0031] With continued reference to FIGS. 1-4, the piston plates 15 may each be secured to the containment housing 10 in a manner that allows movement of the piston plates 15 relative to the containment housing 10 with substantially no bending or flexing of the piston plates 15 during operation of the marine acoustic vibrator 5. In the embodiment of FIG. 1, a pair of piston plates 15 is shown. One of the piston plates 15 may be disposed on one side of the containment housing 10 while the other of the piston plates 15 may be disposed on the opposing side of the containment housing 10. As illustrated, one of the piston plates 15 may be coupled to the containment housing 10 at or near the first surface 40 and the other piston plate 15 may be coupled to the second surface 45. The piston plates 15 may each cover a corresponding one of the openings 50, 55 in the respective first and second surfaces 40, 45 of the containment housing 10. In the illustrated embodiment, the piston plates 15 are coupled to the containment housing 10 by way of rubber seals 20. The rubber seals 20 may not hold the piston plates 15 in place but rather may flex (or otherwise move) to permit movement of the piston plates 15 at their outer edges. In particular embodiments, the piston plates 15 may function as piston

transducers, wherein each of the piston plates **15** moves back forth by actuation of the drivers **35**. Movement of the pistons plates **15** is illustrated on FIGS. **3** and **4** by arrows **100**. In contrast to flexensional-shell type vibrators, the piston plates **15** may not bend or flex in operation, but rather may move back and forth acting against the surrounding water.

[0032] In the illustrated embodiment, spring elements **25** (e.g., in the form of coil springs) are disposed in the marine acoustic vibrator **5** on either side of the fixture **30**. However As best seen in FIG. **4**, pairs of spring elements **25** may be located on either side of the fixture **30** with a first pair of spring elements **25** disposed on one side of the containment housing **10** and a second pair of the spring elements **25** disposed on the opposing side of the containment housing **10**. The spring elements **25** in the first pair may be disposed on opposite sides of the drivers **35** from one another, and the spring elements **25** in the second pair may also be disposed on opposite sides of the drivers **35** from one another. The spring elements **25** may each extend between a corresponding one of the piston plates **15** and the fixture **30**. The spring elements **25** may be coupled to the piston plates **15** and the fixture **30** to exert a biasing action on the piston plates **15**. A wide variety of different spring elements **25** may be used that are suitable for exerting the desired biasing action against the piston plates **15**, including both linear and non-linear springs. In particular embodiments, the spring elements **25** may be any of a variety of different types of springs, including compression springs, torsion springs, or other suitable springs for exerting the desired biasing action. Specific examples of spring elements **25** that may be used include coil springs, flat springs, bow springs, and leaf springs, among others. Suitable spring elements **25** may be constructed from spring steel or other suitable resilient material, such as glass-fiber reinforced plastic (e.g., glass-fiber reinforced epoxy), carbon fiber reinforced plastic, and combinations thereof. In some embodiments, the dimensions, material make-up, and shape of the spring elements **25** may be selected to provide a sufficient spring constant for vibrations within the seismic frequency range of interest when the marine acoustic vibrator **5** is submerged in water at a depth of from about 0 meters to about 300 meters.

[0033] Drivers **35** may be one of a variety of types of drivers, for example electro-dynamic drivers. In some embodiments, the drivers **35** may be “moving coil” or “voice coil” drivers, which may provide the ability to generate very large acoustic energy amplitudes. Other types of drivers **35** may also be used, such as linear servo motors. Although the particular embodiment described herein shows four uni-directional drivers utilized in parallel, embodiments in which one or more bi-directional drivers, embodiments with one or more uni-directional drivers, or embodiments in which more or less than four uni-directional drivers are utilized, are each within the scope of the invention. As best seen in FIGS. **3** and **4**, a pair of the drivers **35** may be coupled to an interior face **90** of one of the piston plates **15** on one side of the containment housing **10** while another pair of the drivers **35** may be coupled to an interior face **90** of another one of the piston plates **15** on an opposing side of the containment housing. The drivers **35** may also be coupled to the fixture **30** and disposed within the marine acoustic vibrator **5**.

[0034] As illustrated, the drivers **35** may each comprise a uni-directional, moving coil driver, comprising an electric coil **105**, transmission element **110**, and magnetic circuitry

115, which may work together to generate a magnetic field. As illustrated, the magnetic circuitry **115** may be connected to the fixture **30**, while the transmission element **110** may connect to the corresponding piston plate **15**. In some embodiments (not illustrated), this arrangement may be reversed (i.e., the magnetic circuitry **115** connects to the corresponding piston plate **15**, while the transmission element **110** connects to the fixture **30**). As illustrated, each transmission element **110** may transfer motion of the corresponding electric coil **105** to the interior face **90** of the corresponding piston plate **15**. When electrical current **I** is applied to the electric coil **105**, a force **F** acting on the electric coil **105** may be generated as follows:

$$F=IIB$$

(Eq. 11)

[0035] Where **I** is the current, **l** is the length of the conductor in the electric coil **105**, and **B** is the magnetic flux generated by the magnetic circuitry **115**. By varying the magnitude of the electrical current and consequently the magnitude of the force acting on the electric coil **105**, the length of the driver stroke may vary. Each driver **35** may provide stroke lengths of several inches—up to and including about 10”—which may allow the marine acoustic vibrator **5** to generate enhanced amplitude acoustic energy output in the low frequency ranges, for example, between about 1 Hz and about 10 Hz when the marine acoustic vibrator **5** is submerged in water at a depth of from about 0 meters to about 300 meters. Often, the magnetic circuitry **115** may comprise permanent magnets, though any device capable of generating a magnetic flux may be incorporated.

[0036] In some embodiments, the fixture **30** suspends the drivers **35** within the containment housing **10**. For example, in the illustrated embodiment, the fixture **30** extends along the major axis of containment housing **10** and may be coupled to either end of the containment housing **10**. The fixture **30** may be circular, square, rectangular, or other suitable cross-section as desired for a particular application. An example of a suitable fixture **30** may include a rod, beam, plate, or other suitable frame for supporting internal components such as the drivers **35** in the containment housing **10**. In particular embodiments, the fixture **30** may be fixed to the containment housing **10** in a manner that prevents movement to prevent undesired contraction of the major axis of the containment housing **10**. In particular embodiments, the piston plates **15** may work in symmetry above and below the fixture **30**. In other words, in some embodiments, the fixture **30** may divide the marine acoustic vibrator **5** into symmetrical halves with respect to at least the piston plates **15**, spring elements **25**, and drivers **35**.

[0037] Turning now to FIG. **5**, a diagram illustrating another embodiment of a marine acoustic vibrator **5** is shown. As illustrated, the marine acoustic vibrator **5** may comprise a containment housing **10**, which may have a first surface **40**. The piston plate **15** may be indirectly coupled to the first surface of the containment housing **10**. While not shown on FIG. **5**, a rubber seal (e.g., rubber seal **20** on FIG. **1**) may couple the piston plate **15** to the first surface **40** of the containment housing **10**. As illustrated, in some embodiments, the piston plate **15** may be substantially circular in shape while, in other embodiments, the piston plate **15** may have a different shape, such as a substantially square or rectangular shape in one embodiment. Similarly, while the containment housing **10** is depicted on FIG. **5** as having a generally square-shaped first surface **40**, the containment

housing 10 may have other suitable configurations for the first surface 40, including substantially, rectangular, circular, or rounded.

[0038] Turning now to FIG. 6, an embodiment of the rubber seal 20 is depicted. In the illustrated embodiment, coupling of the rubber seal 20 to a piston plate 15 is shown. While not shown, the rubber seal 20 may also be coupled to the containment housing 10, for example, to form a water-tight seal between the piston plate 15 and the containment housing 10. In general, the rubber seal 20 may be configured to allow movement of the piston plate 15 while also maintaining the appropriate seal. As illustrated, the rubber seal 20 may have significant curvature (e.g., curved portion 116) to permit a significant amplitude of movement. By way of example, this permitted movement may further enable the piston plates 15 to have several inches of travel, e.g., the piston plates 15 may move back and forth relative to containment housing 10 a distance of from about 1 inch to about 10 inches (or more). While FIG. 6, illustrates the rubber seal 20 with the curved portion 115 to permit movement, other techniques for permitting movement may be used, including use of seals with a bellows or accordion-type configuration. Moreover, while preceding description describes use of rubber, other materials of construction may be used in coupling the piston plates 15 to the containment housing 10 that can provide the desired seal and movement.

[0039] Turning now to FIG. 7, a cross-sectional view of one embodiment of the marine acoustic vibrator 5 is provided that includes alternative spring elements 120. The cross-sectional view is taken along line 3-3 of FIG. 2. In contrast to FIG. 2 which illustrates spring elements 25 in the form of coil springs, FIG. 7 illustrates the alternative spring elements 120 in the form of bow springs. In the cross-sectional view of FIG. 7, certain elements of the marine acoustic vibrator such as the drivers 35 are not visible.

[0040] The following description is for one of the alternative spring elements 120; however, because the fixture 30 provides a line of symmetry, this description is equally applicable to both of the alternative spring elements 120. As illustrated, one of the alternative spring element 120 may be coupled between one of the piston plate 15 and the fixture 30 one side of the containment housing 10. The alternative spring element 120 may be coupled to the piston plate 15 at attachment point 125, which may be a fixed connection, for example, that does not permit movement. The alternative spring element 120 may be coupled on either end to a supplemental fixture 130, which may be in the form of a beam, rod, plate, or other suitable frame for supporting the alternative spring element 120 in the containment housing 10. The alternative spring element 120 may be coupled to the supplemental fixture 130 by way of bearings 135. In particular embodiments, the bearings 135 may permit movement of the spring element 120, for example, linear bearings that permit linear movement of the ends of alternative spring element 120 as represented by arrows 140. In this manner, the alternative spring element 120 may be allowed to flex and provide a biasing force to the piston plate 15 upon its movement. The supplemental fixture 130 may be coupled to the fixture 30 at one or more fixture attachment points 145, which may be fixed connections that do not permit movement.

[0041] Turning now to FIG. 8, another embodiment of a marine acoustic vibrator 5 is illustrated. In the illustrated embodiment, the marine acoustic vibrator 5 further includes

mass springs 150 with weights 155 affixed thereto. The mass springs 150 shown on FIG. 7 may also be used in conjunction with the alternative spring elements 120 shown on FIG. 6 (or other suitable type of spring element). As illustrated, the mass springs 150 may be coupled between the fixture 30 and the piston plates 15. In the illustrated embodiment, a pair of mass springs 150 is shown on either side of the fixture 30 so that the marine acoustic vibrator 5 comprises four mass springs 150. However, it should be understood that more or less than four mass springs 150 may be utilized for a particular application. As will be described below, in various embodiments, the spring constant of the mass springs 150 and the mass of the weights 155 may be selected in a manner to achieve a second system resonance frequency within the seismic frequency range of interest when the marine acoustic vibrator 5 is submerged in water at a depth of from about 0 meters to about 300 meters. In a particular embodiment, the marine acoustic vibrator 5 may exhibit a first resonance frequency of about 2.5 Hz and a second resonance frequency of about 4.5 Hz when submerged in water at a depth of from about 0 meters to about 300 meters. Although a marine acoustic vibrator 5 as shown in FIG. 1 that did not include the mass springs 150 would typically display a second resonance frequency, for systems having a size suitable for use in geophysical exploration, the second resonance frequency when the marine acoustic vibrator 5 shown on FIG. 1 is submerged in water at a depth of from about 0 meters to about 300 meters would typically be much higher than the frequencies within the seismic frequency range of interest.

[0042] In various embodiments, finite element analysis may be utilized in order to appropriately tune the piston plates 15, spring elements 25 (or alternative spring elements 120), mass springs 150, and weights 155. In such an analysis the following principles of operation may be relevant. With the piston plate 15, then, for low frequencies, the mass load, or the equivalent fluid mass acting on the piston plate 15 is:

$$M_{\text{piston_plate}} = \rho_o (8a^3/3) \quad (\text{Eq. 11})$$

where $M_{\text{piston_plate}}$ is the mass load acting on the piston plate 15, ρ_o is the density of water surrounding the marine acoustic vibrator, and a is the equivalent radius for the piston plate 15 which corresponds to the size of the piston plate 15.

[0043] The spring elements 25 (or alternative spring elements 120) may also have a spring constant ($K_{\text{piston_spring}}$) in the direction of the moving electric coils (e.g., electric coil 105 on FIGS. 2-4). Therefore, the first resonance ($f_{\text{resonance-1}}$) of the marine acoustic vibrator 5, due to interaction of the piston plates 15 and the spring elements 25 (or alternative spring elements 120) with the water, may be substantially determined by the following mass spring relationship:

$$f_{\text{resonance-1}} = \frac{1}{2\pi} \sqrt{\frac{K_{\text{piston_spring}}}{M_{\text{piston_plate}}}} \quad (\text{Eq. 12})$$

where $K_{\text{piston_spring}}$ is the spring constant of a spring elements 25 (or alternative spring elements 120), and $M_{\text{piston_plate}}$ is the mass load acting on the piston plate 15.

[0044] To achieve efficient energy transmission in the seismic frequency range of interest, it may be desirable to achieve a second resonance frequency within the seismic frequency range of interest. In the absence of the mass springs 150, the second resonance frequency would occur

when the piston plate 15 has its second Eigen-mode. This resonance frequency, however, is normally much higher than the first resonance frequency, and accordingly, would typically be outside the seismic frequency range of interest. As is evident from the foregoing equation, the resonance frequency will be reduced if the mass load on the piston plates 15 is increased. However, in some embodiments, in order to add sufficient mass to achieve a second resonance frequency within the seismic frequency range of interest, the amount of mass that would need to be added to the piston plates 15 may make such a system less practical for use in marine seismic surveying operations

[0045] Thus, in some embodiments, the mass springs 150 may be included inside the containment housing 10 with the weights 155 on the side of the mass springs 150. In one embodiment, these mass springs 150 may have a transformation factor T_{mass_spring} between the long and short axis of its ellipse, so that the deflection of the two side portions will have a higher amplitude than the deflection of end attached to the piston plate 15 and electric coil.

[0046] The effect of such mass springs 150 may be equivalent to adding mass in the end of the drivers 35 where it is attached to the piston plates 15.

$$M_{mass_spring} = (T_{mass_spring})^2 \cdot M_{added} \quad (\text{Eq. 13})$$

Wherein M_{mass_spring} is the mass of the mass springs 150, T_{mass_spring} is the transformation factor of the mass springs 150, and M_{added} is the mass of the weights 155.

[0047] Use of the mass springs 155, may allow the second resonance frequency of the system to be tuned so that the second resonance frequency is within the seismic frequency range of interest, thereby improving the efficiency of the marine acoustic vibrator 5 in the seismic frequency range of interest.

$$f_{resonance\ 2} = \frac{1}{2\pi} \sqrt{\frac{K_{mass_spring} + K_{piston_spring}}{(T_{spring})^2 \cdot M_{added} + M_{piston_plate}}} \quad (\text{Eq. 14})$$

where K_{mass_spring} is the spring constant of mass springs 150, K_{piston_spring} is the spring constant of spring elements 25 (or alternative spring element 120), T_{mass_spring} is the transformation factor of the mass springs 150, M_{added} is the mass of the weights 155, and M_{piston_plate} is the mass load on the piston plates 15.

[0048] Accordingly, it may be possible, as shown above, to select the weights 155 on the mass springs 150 to tune the second resonance frequency. It may also be possible to select the extent of influence the second resonance frequency may have on the system. By way of example, if the mass springs 150 have a low spring constant compared to the spring elements 25 (or alternative spring elements 120), and a matching of weights 155 is added to the mass springs 150, the mass springs 150 will function relatively independently from the piston plates 15. In such cases, the second resonance frequency may be as follows:

$$f_{resonance\ 2} = \frac{1}{2\pi} \sqrt{\frac{K_{mass_spring}}{(T_{mass_spring})^2 \cdot M_{added}}} \quad (\text{Eq. 15})$$

In the same way, it may also be possible in some embodiments to make the second resonance frequency very large by selecting a high spring constant for the mass springs 150 (e.g., stiffer mass springs 150) with a matching of weights 155 such that the second resonance frequency will have a larger amplitude than the first resonance frequency.

[0049] Turning now to FIG. 9, a diagram is provided illustrating one embodiment of an array 160 of marine acoustic vibrators 5. Depending on the desired acoustic energy output, it may be desired to use an array 160 of marine acoustic vibrators 5, such as those shown in FIGS. 1-3 and 5-7. For example, the two or more marine acoustic vibrators 5 in the array 160 may be used in sequence, contemporaneously, or even simultaneously. As would be understood by one of ordinary skill in the art with the benefit of this disclosure, energy emitted from the two or more marine acoustic vibrators 5 may appear in the rock formations below the water bottom as if it emanated from a point source when the dimensions of the array are on the order of about 30 meters or less. FIG. 9 illustrates two marine acoustic vibrators 5 in the array 160; however, more than two marine acoustic vibrators 5 may be used in the array 160, for example, depending on the desired acoustic energy output.

[0050] As illustrated, the marine acoustic vibrators 5 may each be disposed in a corresponding submersible vessel 165 for towing through a body of water 170. The submersible vessels 165 may be configured to support and position the marine acoustic vibrators 5 in the body of water 170. In some embodiments, the submersible vessels 165 may each include a fin 175, which may be used, for example, to position the corresponding marine acoustic vibrator 5 in the body of water 170. As illustrated, the submersible vessels 165 may each include a bracket 180 or other suitable attachment point for coupling the marine acoustic vibrators 5 to a corresponding lead line 185. In the illustrated embodiment, the array 160 may be used in conjunction with at least one hydrodynamic depressor 190 that may create thrust as moved through the body of water 170 to force the array 160 downward into the body of water 170. In some embodiments, the towing configuration may provide for towing the marine acoustic vibrators 5 at different depths. In some embodiments, the marine acoustic vibrators will be held apart by a support structure (not shown). When utilized, the support structure may couple the marine acoustic vibrators 5 to one or more lead lines 185. Such variations in towing configuration should be apparent to one of ordinary skill in the art with the benefit of this disclosure.

[0051] FIG. 9 illustrates an example technique for acquiring marine seismic data that can be used with embodiments of the present techniques. In the illustrated embodiment, a survey vessel 200 moves along the surface of the body of water 170, such as a lake or ocean. The survey vessel 200 may include thereon equipment, shown generally at 205 and collectively referred to herein as a "recording system." The recording system 205 may include devices (none shown separately) for detecting and making a time indexed record of signals generated by each of seismic sensors 210 (explained further below) and for actuating one or more sound sources (as illustrated, marine acoustic vibrator 5 and/or air guns 215) at selected times. The recording system 205 may also include devices (none shown separately) for determining the geodetic position of the survey vessel 200 and the various seismic sensors 210.

[0052] As illustrated, the survey vessel **200** (or a different vessel) may tow the marine acoustic vibrator **5** in the body of water **170**. A source cable **220** may couple the marine acoustic vibrator **5** to the survey vessel **200**. The marine acoustic vibrator **5** may be towed in the body of water **170** at a depth ranging from 0 meters to about 300 meters, for example. While only a single marine acoustic vibrator **5** is shown in FIG. 9, it is contemplated that embodiments may include more than one marine acoustic vibrator **5**, or one or more arrays thereof (e.g., array **160** on FIG. 8). Optionally, the survey vessel **200** may further tow one or more air guns **215** (or other sound sources) by the same source cable **220** or different source cable **225**. As illustrated, the air guns **215** may be towed at a different depth than the marine acoustic vibrator **5**. While FIG. 9 illustrates the air guns **215** being towed at a shallower depth than the marine acoustic vibrator **5**, it is contemplated that the air guns **215** may be towed at the same or a greater depth than the marine acoustic vibrator **5**.

[0053] At selected times, the marine acoustic vibrator **5** (and/or the air guns **215**) may be triggered, for example, by the recording system **205**, to generate acoustic energy. The survey vessel **200** (or a different vessel) may further tow at least one sensor streamer **230** to detect the acoustic energy that originated from the marine acoustic vibrator **5** (and/or the air guns **215**) after it has interacted, for example, with rock formations **235** below the water bottom **240**. As illustrated, the marine acoustic vibrator **5**, the air guns **215**, and the sensor streamer **230** may be towed above the water bottom **240**. The sensor streamer **230** may contain seismic sensors **210** thereon at spaced apart locations. While not shown, some marine seismic surveys locate seismic sensors **210** on ocean bottom cables or nodes in addition to, or instead of, a sensor streamer **230**. The seismic sensors **210** may be any type of seismic sensors known in the art, including hydrophones, geophones, particle velocity sensors, particle displacement sensors, particle acceleration sensors, or pressure gradient sensors, for example. By way of example, the seismic sensors **210** may generate response signals, such as electrical or optical signals, in response to detected acoustic energy. Signals generated by the seismic sensors **210** may be communicated to the recording system **205**. In some embodiments, more than one sensor streamer **230** may be towed by the survey vessel **200**, and the sensor streamers **230** may be spaced apart laterally, vertically, or both laterally and vertically. The detected energy may be used to infer certain properties of the subsurface rock, such as structure, mineral composition, and fluid content, thereby providing information useful in the recovery of hydrocarbons.

[0054] In accordance with an embodiment of the invention, a geophysical data product may be produced. The geophysical data product may include data that is obtained by a process that includes detecting the acoustic energy originating from the marine acoustic vibrator **5**. The geophysical data product may be stored on a non-transitory, tangible computer-readable medium. The geophysical data product may be produced offshore (i.e. by equipment on a vessel) or onshore (i.e. at a facility on land) either within the United States or in another country. If the geophysical data product is produced offshore or in another country, it may be imported onshore to a facility in the United States. Once

onshore in the United States, geophysical analysis, including further data processing, may be performed on the geophysical data product.

[0055] Although specific embodiments have been described above, these embodiments are not intended to limit the scope of the present disclosure, even where only a single embodiment is described with respect to a particular feature. Examples of features provided in the disclosure are intended to be illustrative rather than restrictive unless states otherwise. The above description is intended to cover such alternatives, modifications, and equivalents as would be apparent to a person skilled in the art having the benefit of this disclosure.

[0056] The scope of the present disclosure includes any feature of combination of features disclosed herein (either explicitly or implicitly), or any generalization thereof, whether or not it mitigates any or all of the problems addressed herein. Various advantages of the present disclosure have been described herein, but embodiments may provide some, all, or none of such advantages, or may provide other advantages.

What is claimed is:

1. A system comprising:
 - a marine acoustic vibrator comprising a containment housing, one or more piston plates coupled to the containment housing, and a driver coupled to the one or more piston plates to move the one or more piston plates back and forth; and
 - a submersible vehicle that supports and positions the marine acoustic vibrator in a body of water.
2. The system of claim 1, further comprising a lead line coupled to an attachment point on the submersible vehicle.
3. The system of claim 1, further comprising a hydrodynamic depressor used in conjunction with the marine acoustic vibrator for creating thrust as the hydrodynamic depressor is moved through the body of water to force the marine acoustic vibrator downward.
4. The system of claim 1, wherein the submersible vehicle comprises a fin.
5. The system of claim 1, wherein the marine acoustic vibrator is disposed in a window of the submersible vehicle.
6. The system of claim 1, further comprising a source cable coupling the marine acoustic vibrator to a survey vessel.
7. The system of claim 1, wherein the one or more piston plates each cover a corresponding opening in the containment housing, the one or more piston plates being coupled to the containment housing using rubber seals.
8. The system of claim 1, wherein the one or more piston plates comprise a pair of piston plates being disposed on opposing sides of the containment housing from one another.
9. The system of claim 1, further comprising a fixture coupled to the containment housing and spring elements that extend between the fixture and the one or more piston plates.
10. The system of claim 9, wherein the one or more piston plates comprise a first piston plate and a second piston plate on an opposing side of the containment housing from the first piston plate, wherein the spring elements comprise a first pair of spring elements coupled to the fixture and the first piston plate and a second pair of spring elements coupled to the fixture and the second piston plate.
11. The system of claim 10, wherein the fixture provides a line of symmetry that divides the marine acoustic vibrator

in symmetrical halves with respect to at least the one more piston plates, the spring elements, and the driver, wherein the fixture comprises a frame.

12. The system of claim **9**, wherein the fixture supports the driver in the containment housing, the extending along a major axis of the containment housing, wherein the fixture is coupled to either end of the containment housing.

13. The system of claim **1**, wherein the one or more piston plates are configured to not bend or flex during operation of the marine acoustic vibrator.

14. The system of claim **1**, wherein the marine acoustic vibrator is configured to have a resonance frequency within a frequency range of about 1 Hz and about 10 Hz when disposed in water at a depth of about 0 meters to about 300 meters.

15. A method comprising:

towing a submersible vehicle through a body of water, wherein a marine acoustic vibrator is disposed in the submersible vehicle; and

triggering the marine acoustic vibrator to cause one or more piston plates in the marine acoustic vibrator to move back and forth and generate acoustic energy, wherein one or more spring elements exert a biasing force against the one or more piston plates.

16. The method of claim **15**, wherein the marine acoustic vibrator generates at least one resonance frequency within a frequency range of about 1 Hz and about 10 Hz.

17. The method of claim **15**, further comprising towing an array of marine acoustic vibrators in the body of water, each of the marine acoustic vibrators being disposed in a corresponding submersible vehicle, the array of marine acoustic vibrators comprising the marine acoustic vibrator.

18. The method of claim **17**, further comprising generating downward thrust with a hydrodynamic depressor to force the array of marine acoustic vibrators downward in the body of water.

19. The method of claim **15**, further comprising towing one or more air guns in conjunction with the marine acoustic vibrator, and triggering the one or more air guns to generate acoustic energy.

20. The method of claim **17**, further comprising: obtaining geophysical data; processing the geophysical data to generate a geophysical data product, and recording the geophysical data product on a tangible, non-volatile computer-readable medium suitable for importing onshore, wherein the geophysical data is obtained by a process that includes detecting the acoustic energy originating from the marine acoustic vibrator.

* * * * *