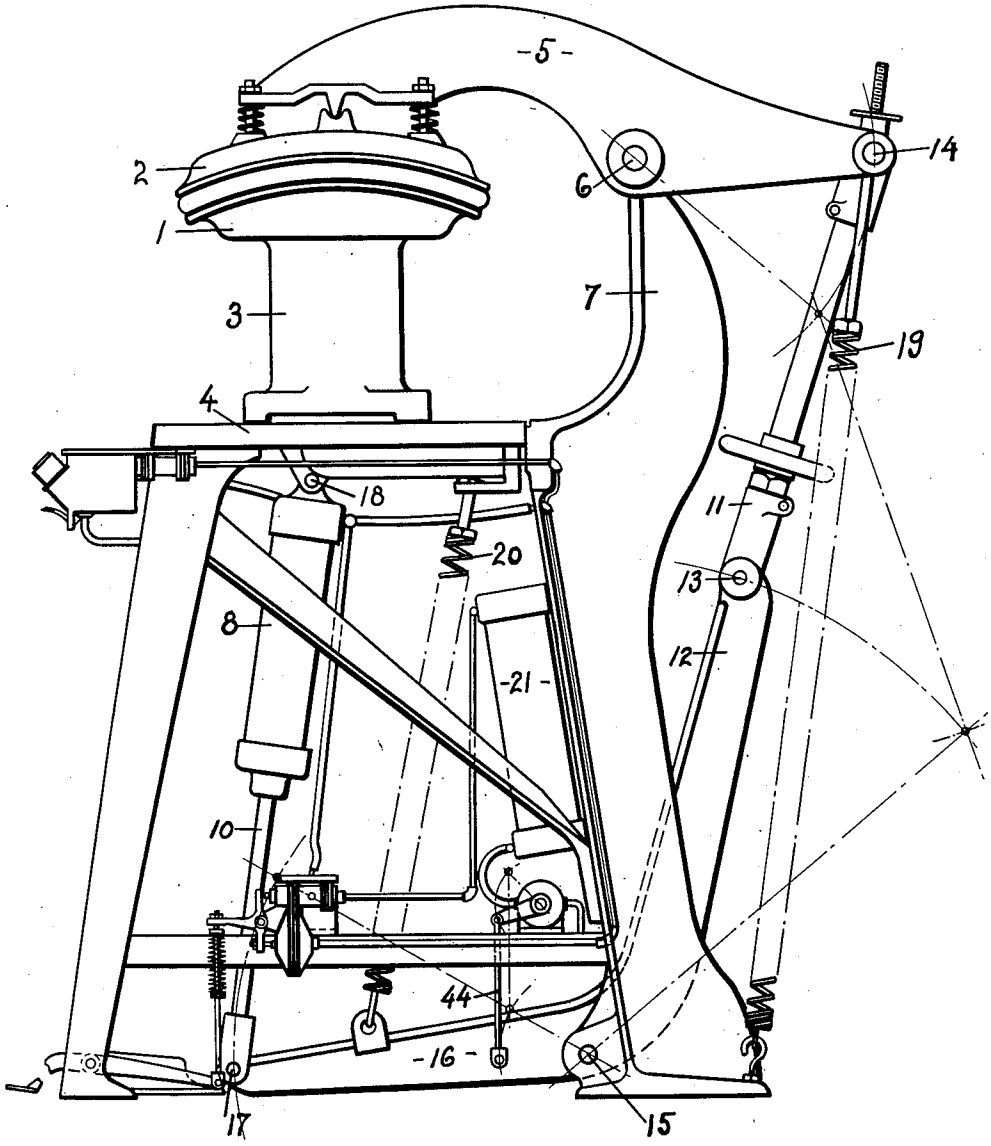
Sept. 4, 1934. E. DAVIS 1,972,474

GARMENT OR IRONING PRESS

Original Filed Sept. 27, 1929 3 Sheets-Sheet 1

Fig. 1.



INVENTOR.

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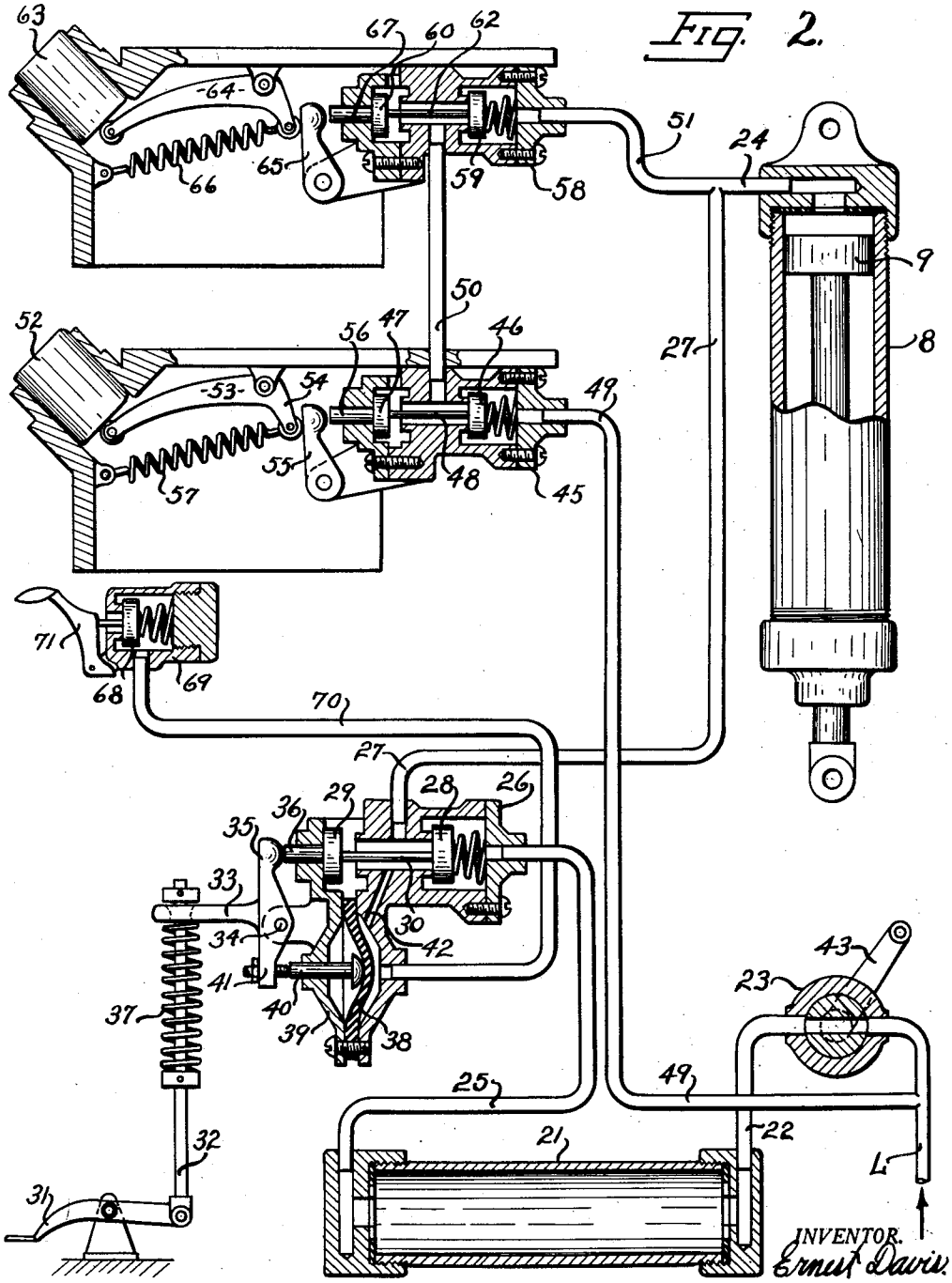
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GARMENT OR IRONING PRESS

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Fig. 2.



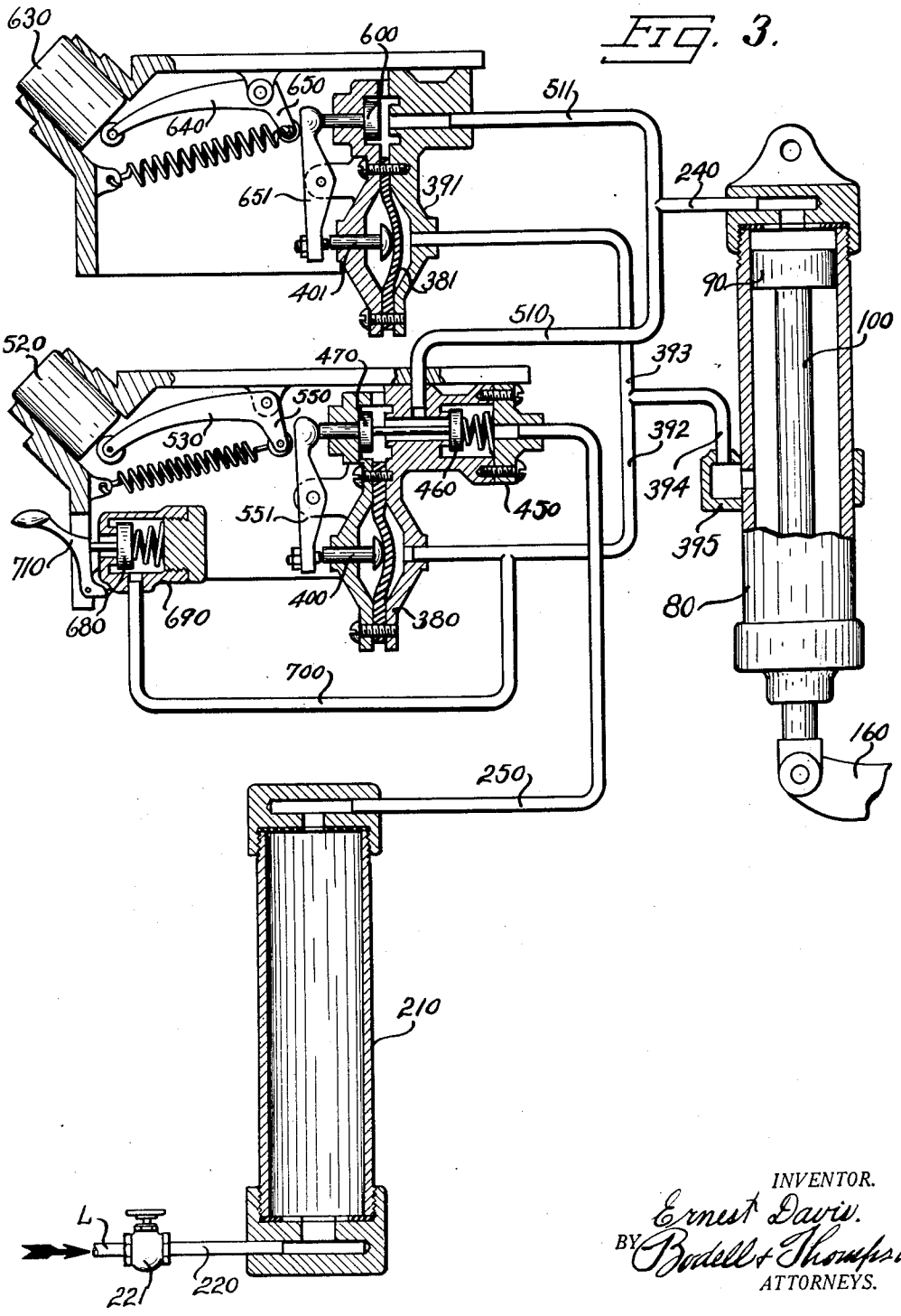
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FIG. 3.



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UNITED STATES PATENT OFFICE

1,972,474

GARMENT OR IRONING PRESS

Ernest Davis, Syracuse, N. Y., assignor to The Prosperity Company Inc., Syracuse, N. Y., a corporation of New York

Application September 27, 1929, Serial No. 395,720
Renewed September 14, 1932

19 Claims. (Cl. 68—9)

This invention relates to pressing machines as garment and laundry or ironing pressing machines and has for its object a particularly simple and efficient motor means and controls therefor by which the press is preliminarily closed under light pressure, or so-called zero pressure and may be closed at the will of the operator under a graduated degree of final pressure suiting the will of the operator.

An object of the invention is to devise a new and novel control and operating mechanism for garment or laundry presses whereby the press is first preliminarily closed under light or zero pressure and subsequently is closed under final heavy pressure.

Another object is to use a temporary air supply of predetermined and limited capacity to preliminarily close the press and thereafter to close the press under increased or final heavy pressure.

The invention consists in the novel features and in the combinations and constructions hereinafter set forth and claimed.

In describing the invention reference is had to the accompanying drawings in which like characters represent corresponding parts in all the views.

Fig. 1 is a side elevation of a pressing machine embodying my invention.

Fig. 2 is a diagrammatic sectional view of the motor and the controls therefor.

Fig. 3 is a view similar to Fig. 2 of a modified form of one feature of my invention omitting other features included in Fig. 2.

This invention comprises generally, means for controlling the initial flow of a predetermined or measured amount of motive fluid to the motor means of the press to actuate the motor sufficiently to close the press or press jaws and check or stop the closing movement and then close the press under final pressure.

The invention is here illustrated as embodied in a conventional type of garment or laundry press in which the cooperating pressing elements or jaws are respectively a buck 1 and a head 2 movable toward and from the buck. The buck is mounted on a suitable bolster or bracket 3 which in turn is mounted on a suitable frame 4. The head is carried by a suitable frame or yoke lever 5 pivoted between its ends at 6 to a standard 7 rising from the frame.

The motor means is preferably a single cylinder and piston unit and is operated by fluid pressure as compressed air. The motor here shown includes a cylinder 8 having a piston 9 therein, the piston rod 10 of which is connected through suitable motion transmitting mechanism to the head 2. This mechanism in the illustrated embodiment of my invention, in addition to the lever 5, comprises toggle links 11 and 12 pivoted together at 13, and at their opposite ends, respectively at 14 and 15, to the rear arm of the lever 5 and to the base of the frame 4. The link 12 has a forwardly extending arm 16 which forms a bell-crank lever with the link 12. The arm 16 is pivoted at 17 to the piston rod 10 and the cylinder 8 is pivoted at 18 to the frame so that it may oscillate. The piston 9 is single acting and the press is opened, when the pressure in the cylinder 8 is released, by suitable counter-springs 19 and 20. The piston is actuated by a two step movement in which the first step preliminarily closes the press, that is, closes the press or press jaws under light or zero pressure, and the second step closes the press to final pressure from zero pressure position. It is to be understood that when the press closes under light or zero pressure, a predetermined or limited quantity of fluid under pressure is fed to the motor 8 so that that quantity or volume of fluid expands to force the piston 9 outwardly far enough to bring the head close to the buck, or just contacting the buck under zero pressure, or engaging the buck with a light pressure substantially less than the full pressure delivered by the motor. Preliminary closed position of the press jaws 1 and 2 is therefore a position which is spaced from finally closed position of the jaws, in which latter position the jaws are under substantial pressure.

In order to prevent the actuation of the piston beyond zero pressure position without checking the movement or pausing at zero pressure position, means is provided for supplying a predetermined amount of motive fluid, in this case compressed air, to the cylinder sufficient only to bring the press head or jaw down close to, or against the work on the buck 1 with very light pressure. This means is here shown as a tank 21 of such capacity so as to supply only enough motive fluid, or compressed air, to the cylinder 8 to actuate the piston 9 to preliminarily close the press. This tank 21 has an inlet 22 which is connected to a suitable source of compressed air and this inlet, as shown in Fig. 2, is controlled by a normally open valve 23. The tank 21 is connected by a conduit including the pipes 25 and 27 to the intake port 24 of the cylinder 8. This conduit has an operator operated or manually operated intake valve therein which is normally closed.

The conduit 25 lies between or connects the tank 21 and a valve casing 26. The conduit 27 connects the valve casing 26 and the port 24 at the intake end of the cylinder 8. A normally closed spring pressed intake valve 28 and a normally open exhaust valve 29 is within the casing 26, these valves being connected together by a stem 30 so that they operate as a unit. The pipe 25 connects the intake end of the valve casing 26 to the tank 21. The pipe 27 is connected to the

valve casing 26 between the valves 28 and 29. The valves are actuated by suitable operator operated means, here illustrated as a pedal 31 connected by a link 32 and a spring 37 to a lever 33 pivoted at 34 to the valve casing 26 and having an angular arm 35 which thrusts against a stem 36 of the exhaust valve 29. The spring 37 yieldingly applies the pressure of the pedal 31 to the lever 33.

The valves 28 and 29 are locked in their operated positions by the action of a diaphragm 38 located in a diaphragm chamber 39 carried by the valve casing 26. The diaphragm acts against a sliding stem 40 which thrusts against an angular arm 41 of the lever 33. The pressure side of the diaphragm chamber is connected by a duct 42 to the interior of the valve casing 26.

Upon depression of the pedal 31 the intake valve 28 will be opened against the action of its spring and the exhaust valve 29 closed so that air will flow from the tank 21 to the valve casing 26 through the pipe 25 and from the valve casing through the pipe 27 to the cylinder 8 to actuate the piston 9 therein. Air will also flow through the duct 42 to lock the valves 28 and 29 in their operated positions. The tank 21 is of sufficient capacity to actuate piston 9 far enough to bring down the head 2 on the work with little or no pressure. Means is provided for closing the valve 23 at the intake end of the tank 21 when the press is in, or nearly in, preliminarily closed position when the press jaws are under little or no pressure or at any time prior thereto. In order that the head may stop under little or no pressure and not apply final pressure by the building up of the pressure in the cylinder 8, the valve 23 is closed during the closing of the press and opened during the opening of the press, by connections between the movable member of the valve and the motion transmitting mechanism between the piston 9 and the head 2, that is, by any part actuated by the piston 9. The valve 23 is here illustrated as a rotary valve having a rock arm 43 which is connected by a link 44 (see Figure 1) to the arm 16.

When the press is open, the valve 23 is open so that the tank 21 is open to the fluid pressure in the feed line connected with the fluid supply. When the press is closed under preliminary pressure upon the depression of the pedal 31, the valve 23 closes as the press closes so that only a predetermined or measured amount of air flows to the cylinder 8 to actuate the piston therein, this amount being sufficient only to close the press into preliminarily closed position under little or no pressure. When the intake valve 28 is opened and the exhaust valve 29 closed by the depression of the pedal 31, these valves are held in their operated positions by the air passing from the interior of the valve chamber 26 and through the duct 42 to the pressure side of the diaphragm chamber 39 to actuate the diaphragm 38 therein and thus through the stem 40 hold the lever 33 in its operated position. As the valve 23 closes and hence cuts off the tank 21 from the main air line, the movement or the speed of movement of the head is checked somewhat as the head approaches its preliminarily closed position because of the drop in pressure against the piston as it moves towards closed press position. The size of the tank may be of any desired proportion to that of the cylinder 8. The closing of the valve 23 is timed so that the pressure upon the piston 9 in the cylinder 8 will not move the head 2 further than preliminarily closed position.

The means for controlling the closing of the head under heavy pressure and under various or graduated degrees of final pressure, comprises a valve casing 45 having a normally closed spring pressed intake valve 46 and a normally open exhaust valve 47 therein, these valves being connected by a stem 48 so that they act as a unit. The intake end of the valve casing 45 is connected by a pipe 49 to the main feed line L and the valve casing 45 is connected at a point between the valves 46 and 47 to the intake port 24, of the cylinder 8, through pipes 50 and 51 and a valve casing, to be presently described, between the said pipes. The valves 46 and 47 are operated by any suitable manual means, as a push button 52, which acts on a lever 53 having an arm 54 pressing against an intermediate lever 55 thrusting against a stem 56 of the exhaust valve 47. A returning spring 57 is connected to the lever 53.

Depression of the push button 52, closes the exhaust valve 47 through the levers 53 and 55 and opens the intake valve 46 so that air from the main feed line flows to the cylinder 8 and actuates the piston 9 into final pressure position. The operation of the push button 52 will be ineffective unless the pedal 31 has been operated, that is, unless the head has been first brought down into preliminarily closed position, because if the pedal 31 is not first operated, the air will exhaust out through the pipe 27, valve casing 26 and the port controlled by the exhaust valve 29.

In order that both hands of the operator may be required to close the press under final pressure, a second valve and push button is employed. The second valve casing 58 connects the pipes 50 and 51 and includes a check valve 59 and an exhaust valve 60, these being connected by a stem 62 so that they act as a unit. The pipe 50 connects the valve casings 45 and 58 in series and the pipe 51 connects the casing 58 to the intake port 24 of the cylinder 8. A push button 63 acts through levers 64 and 65, against the action of a returning spring 66, to move a stem 67 of the exhaust valve 60 and operate the valves 59 and 60.

Obviously unless both push buttons 63 and 52 are depressed no air will flow to the cylinder 8 from the main line through the pipes 49, 50 and 51. Upon releasing the push buttons the press will be locked owing to the fact that the check valve 59 seats. Repeated depressions of the push buttons 63 and 52 cause the pressure to be increased each time the push buttons are depressed until the fluid pressure in the cylinder equals the pressure of the fluid supply.

The press is opened by releasing the pressure in the diaphragm chamber 39. This is effected by opening a normally closed spring pressed release valve 68 located in a casing 69 which is connected by a pipe 70 to the pressure side of the diaphragm chamber 39. The valve 68 is operated by a manual or trip lever 71 grouped with one of the push buttons 52 or 63 and is here shown grouped with the push button 52. The press may also be opened by depressing the push button 63 in which case the air exhausts from the cylinder 8 through pipe 51, valve casing 58, pipe 50 and out through the open exhaust valve 47 until the pressure in the cylinder is reduced sufficiently to exhaust the air in the diaphragm chamber through the pipe 27, valve casing 26 and duct 42.

In Fig. 3 a pressing machine actuating and control mechanism is shown in which the press head closes with a two step movement, as in the construction shown in Fig. 2, but the press head momentarily slows down or checks instead of stopping as it approaches the buck 2. The air

pressure to the cylinder 80 is supplied from a pressure tank 210 connected to the main line L. In Fig. 3, the cylinder 80 has a piston 90 therein which is carried upon the piston rod 100. A power arm 160 connects the toggle mechanism to the lever carrying the press head. The pressure tank 210 is connected to the feed line L through a pipe 220, the tank 210 being of sufficient capacity to close the press into preliminary closed position.

The intake pipe 220 is always open and it may have an adjustable valve 221 which is preliminarily set in any desired position of restricted opening depending on conditions such as the pressure in the feed line, the checking action desired, etc. The valve 221 restricts or retards the flow of air into the tank 210. The tank 210 at the start of the operation has sufficient capacity to close the press head under light pressure when the push buttons, to be presently described, are operated. that is bring the head 2 down with a decelerating movement as it approaches the buck. This result is obtained because the piston moves rapidly under the initial pressure in the tank 210, which pressure decreases as the piston moves to preliminary press closed position because the inflow of air from the feed line L is restricted by the valve 221. Thereafter when the piston has stopped moving or is moving slowly, the pressure within the tank 210 and cylinder 80 builds up from the feed line L through the restricted inlet in valve 221, and applies heavy or final pressure through the cylinder 80 and the piston 90 to the press head. Thus the head is momentarily checked in its closing movement and does not close with a violent or slamming action.

In Fig. 3 the tank 210 is connected by a pipe 250 to the intake end of a valve casing 450 having a normally closed spring pressed intake valve 460 and a normally open exhaust valve 470 therein connected together, so that they act as a unit. These valves are operated by a push button 520 acting through levers 530, 550 and 551. The valve casing 450 is connected to the intake port of the cylinder 80 through a pipe 510. Obviously, depression of the push button 520 will open the intake valve 460 so that air enters the cylinder 80 to actuate the piston therein.

In order that both hands of the operator must be employed to close the press, a second push button 630 is used which through levers 640, 650 and 651 serves to close a normally open exhaust valve 600 in a valve casing 580 which casing is connected by a pipe 511 to the intake port 240 of the cylinder. It is obvious that unless the push button 630 is operated to close the exhaust valve 600 when the push button 520 is operated, the air instead of passing into the cylinder will pass out through the pipe 511 and the valve casing 580, and that upon the operation of the push button 630 only, no air flows because the intake valve 460 remains closed.

The valves operated by the push buttons 520 and 630 are locked in their operated position in the following manner. Diaphragms 380 and 381 are located in diaphragm chambers 390 and 391 respectively, and these diaphragms act upon the sliding stems 400 and 401 respectively, which in turn act upon arms of the intermediate levers 551 and 651 respectively. The pressure sides of the diaphragm chambers 390 and 391 are connected by pipes 392 and 393 to a pipe 394 extending along the side of the cylinder 80. The pipe 394 communicates with a casing 395 having a port opening into the cylinder, this port being located so as to be uncovered by the piston when the piston approaches the end of its power stroke. Obviously, when the piston does uncover the port, the air will pass back through the casing 395, pipe 394 and pipes 392 and 393 to the diaphragm chambers 390 and 391 and actuate the diaphragms therein to lock the valves in the valve casings 450 and 580 in their operated positions.

To open the press shown in Figure 3, the air is exhausted from the diaphragm chambers 390 and 391 by opening a normally closed exhaust release valve 680 in a casing 690 which is connected by a pipe 700 and by the pipes 392 and 393 to the diaphragm chambers 390 and 391. The release valve 680 is operated by a suitable trip lever 710 grouped with the push button 520.

In both of the illustrated embodiments of my invention, the air is supplied to the cylinder from a tank of sufficient capacity to preliminarily close the press head down lightly with a checking or retarding action as the head 1 approaches the buck 2.

In the form of the invention shown in Fig. 2, the head will remain in preliminary closed position as long as desired or until the operator actuates the push buttons 63 and 52. While the press head is closed in light or no pressure position, steaming, air blowing or sucking operations may be performed, it being understood that the machine is provided and connected with a supply of steam to heat the head or buck or both, and in garment presses to spray steam on the work, and also may be provided with means for blowing or sucking air through the work in the conventional manner.

In the form shown in Fig. 3, the pressing head pauses or slows up automatically as it approaches the buck and then goes gradually under final pressure as the pressure builds up in the tank 210 due to motive fluid entering the tank and cylinder 80 through the open but restricted air inlet provided by the valve 221.

In both forms of the invention, as shown in Figures 2 and 3, the tanks 21 and 210 are energy storing devices in which each tank has a capacity for storing compressed air or energy sufficient to move the operating mechanism from open press position to a preliminary closed position. Even though the tank 210 is always connected with the fluid feed line L, the valve 221 in having a restricted flow therethrough, provides for all practical purposes a cut off which renders only the fluid in the tank 210 available for preliminary closing of the press. The fluid pressure from the feed line L cannot pass through valve 221 fast enough to assist to any material extent in causing preliminary closing of the press which must therefore be caused by the stored air or energy in the tank energy storing device 210.

What I claim is

1. In a garment or laundry pressing machine, cooperating pressing elements, one of which is movable toward and from the other, a motor unit, motion transmitting mechanism between the motor unit and the movable element, means operable to automatically admit a predetermined amount of motive fluid to the motor unit to actuate the motor to close the press and additional means for controlling the flow of additional motive fluid to the motor unit to close the press under heavy pressure.

2. In a garment or laundry pressing machine, cooperating pressing elements, one of which is movable toward and from the other, a motor unit, motion transmitting mechanism between the mo-

tor unit and the movable element, operator controlled means for initially supplying a measured amount of motive fluid to the motor to preliminarily close the press, and operator operated means for controlling the flow of motive fluid to the motor to apply final heavy pressure to the pressing elements.

3. In a garment or laundry pressing machine, cooperating pressing elements, one of which is movable toward and from the other, a motor, motion transmitting mechanism between the motor and the movable element, operator controlled means for initially supplying a measured amount of motive fluid to the motor to preliminarily close the press, operator operated means for controlling the flow of motive fluid to the motor to apply final heavy pressure to the pressing elements, and means for locking the press closed in preliminarily closed position and under final heavy pressure.

4. In a pressing machine, cooperating pressing elements, one of which is movable toward and from the other, a pressure operating motor, motion transmitting connections between the motor and the movable element, a pressure tank, operator operated valve means for controlling the flow of motive fluid from the tank to the motor, said tank having an inlet for connection to a source of supply of motive fluid, a normally open valve in said inlet, means operated by the closing of the press to close said valve, and additional operator operated valve means for controlling the flow of motive fluid to the motor independently of said tank.

5. In a pressing machine, cooperating pressing elements, one of which is movable toward and from the other, a pressure operated motor, motion transmitting connections between the motor and the movable element, a pressure tank, operator operated valve means for controlling the flow of motive fluid from the tank to the motor, said tank having an inlet for connection to a source of supply of motive fluid, a normally open valve in said inlet, means operated by the closing of the press to close said valve, additional operator operated valve means for controlling the flow of motive fluid to the motor independently of said tank, and means controlled by the first mentioned valve means for preventing the operation of the motor by the second mentioned valve means until the first mentioned valve means has been operated to preliminarily close the press.

6. In a pressing machine, cooperating pressing elements, one of which is movable toward and from the other, a pressure operated motor, motion transmitting connections between the motor and the movable element, a pressure tank, operator operated valve means for controlling the flow of motive fluid from the tank to the motor, said tank having an inlet for connection to a source of supply of motive fluid, a normally open valve in said inlet, means operated by the closing of the press to close said valve, additional operator operated valve means for controlling the flow of motive fluid to the motor independently of said tank, means controlled by the first mentioned valve means for preventing the operation of the motor by the second mentioned valve means until the first mentioned valve means has been operated to preliminarily close the press, and means for locking the press closed.

7. In a pressing machine, cooperating pressing elements, one of which is movable toward and from the other, a pressure operated motor having a movable member, motion transmitting connections between the movable motor member and the movable pressing element, a tank for a motive fluid and having an inlet connected to a source of supply and an outlet conduit connected to the motor, an inlet valve in the inlet, a valve casing connected in said outlet conduit, operator operated valve means in said casing, the valve casing having an outlet port controlled by the valve means, a second valve casing, an additional operator operated means therein, the second casing connected to the motor and to a source of supply independently of the tank, means for locking the first mentioned valve means in its operated position, and means operated by the preliminarily closing movement of the press for closing the inlet valve of said tank.

8. In a garment or laundry pressing machine, cooperating pressing elements, one of which is movable toward and from the other, a pressure operated motor, motion transmitting mechanism between the motor and the movable element, a tank having a predetermined capacity for motive fluid relatively to the capacity of the motor so that the motive fluid in the tank is just sufficient to close the press under light pressure, an intake for motive fluid connected to the tank, a conduit connecting the tank and the motor, and valve means in said conduit for controlling the flow of motive fluid from the tank to the motor.

9. In a pressing machine, cooperating pressing elements, one of which is movable toward and from the other, a pressure operated motor, motion transmitting mechanism between the motor and the movable element, a tank having a predetermined capacity for motive fluid relatively to the capacity of the motor so that the motive fluid in the tank is just sufficient to close the press under light pressure, an intake for connection to a source of supply of motive fluid connected to the tank, a conduit connecting the tank and the motor, operator operated valve means for controlling the flow of motive fluid from the tank to the motor, means controlled by the closing of the press for locking the valve means in its operated position, and means for releasing the valve means to open the press.

10. In a garment or laundry pressing machine, in combination, cooperative pressing elements one of which is movable toward and from the other to close and open the press; operating mechanism for the movable pressing element; an energy storing device with capacity only sufficient for the energy required to move the operating mechanism from open-press position to a preliminary closed position; and means connecting the energy storage device with the operating mechanism.

11. In a garment or laundry pressing machine, in combination, cooperative pressing elements one of which is movable toward and from the other to close and open the press; actuating mechanism for the movable pressing element; gaseous fluid operated cylinder and piston motor means for operating the actuating mechanism; a storage device for working fluid under pressure and of such capacity with respect to the motor means that the working fluid stored therein is equal to the amount required to move the actuating mechanism from open-press position to preliminary closed press position under initial light pressure; and means connecting the working fluid storage device with the motor means.

12. In a garment or laundry pressing machine, in combination, cooperative pressing elements one of which is movable toward and from the other to close and open the press; actuating mech-

anism for the movable pressing element; motor means for operating the actuating mechanism; an energy storage device; means connecting the energy storage device with the motor means; means for releasing the energy in said energy storage device to operate the motor to bring the pressing elements together under light pressure; and means for applying additional energy to the motor means to operate the motor means to apply final heavy pressure to the pressing elements.

13. In a garment or laundry pressing machine, in combination, cooperative pressing elements one of which is movable toward and from the other to close and open the press; actuating mechanism for the movable pressing element; fluid operated motor means for operating the actuating mechanism; a working fluid storage device; means connecting the storage device with the motor means; means for releasing the working fluid in said storage device to operate the motor to bring the pressing elements together under light pressure; and means operated by the actuating mechanism for controlling the recharging of the energy storage device.

14. In a garment or laundry pressing machine, in combination, cooperative pressing elements one of which is movable toward and from the other; actuating mechanism for the movable pressing element; fluid motor means for operating the actuating mechanism; a tank of a predetermined capacity relative to the motor means for storage of working fluid under pressure for the motor means; a conduit between the tank and the motor means; a valve in the conduit; operator-operated control means for opening the valve to admit working fluid from the tank to the motor means; and independent control means for supplying working fluid from a main supply line to the motor means.

15. In a garment or laundry pressing machine, in combination, cooperative pressing elements one of which is movable toward and from the other; actuating mechanism for the movable pressing element; fluid motor means for operating the actuating mechanism; a tank for storage of a measured amount of working fluid to operate the motor means a predetermined distance; a conduit between the tank and the motor means; a valve in the conduit; operator-operated control means for the valve; independent control means for supplying working fluid from a main supply line to the motor means; and automatically operated valve means for supplying working fluid from a main supply line to recharge the storage tank.

16. A garment or ironing press comprising, cooperating pressing jaws one of which is movable towards and from the other to close and open the press, actuating mechanism for the movable pressing jaw, fluid motor means operatively connected with the actuating mechanism, means supplying only a sufficient quantity of fluid under pressure to the fluid motor means so that the expansion of this quantity of fluid in the motor means will close the press to preliminary closed position which is spaced from finally closed position, and operator actuated means controlling the connection of the aforesaid means with the motor means and controlling the application of additional fluid pressure to the motor means.

17. A garment or ironing press comprising, cooperating pressing jaws one of which is movable towards and from the other to close and open the press, actuating mechanism for the movable pressing jaw, fluid motor means operatively connected with the actuating mechanism, means including a tank supplying only a sufficient quantity of fluid under pressure to the fluid motor means so that expansion of this quantity of fluid in the motor means will close the press to preliminary closed position which is spaced from finally closed position, and operator actuated means controlling the connection of the aforesaid means with the motor means and controlling the application of additional fluid pressure to the motor means.

18. A garment or ironing press comprising, cooperating pressing jaws one of which is movable towards and from the other to close and open the press, actuating mechanism for the movable pressing jaw, fluid motor means operatively connected with the actuating mechanism, means supplying only a sufficient quantity of fluid under pressure to the fluid motor means so that the expansion of this quantity of fluid in the motor means will close the press into preliminary closed position which is spaced from finally closed position, operator actuated means controlling the connection of the aforesaid means with the motor means, and means controlling the application of an increased fluid pressure to the motor means.

19. A garment or ironing press comprising, cooperating pressing jaws one of which is movable towards and from the other to close and open the press, actuating mechanism for the movable pressing jaw, fluid motor means operatively connected with the actuating mechanism; means supplying only a sufficient quantity of fluid under pressure to the fluid motor means so that the expansion of this quantity of fluid in the motor means will close the press into preliminary closed position including a tank having a predetermined capacity; operator actuated means controlling the connection of the aforesaid means with the motor means, and means controlling the application of full line pressure to the motor means.

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